

MECHATRONIC DESIGN

Lab 2: Introduction to Finite Element (FE) method using CALFEM toolbox (static analysis)

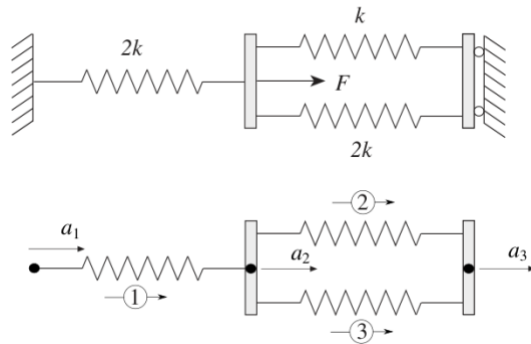
Install CALFEM Toolbox and test its conjunction with MATLAB <https://github.com/CALFEM/calfem-matlab>

The general procedure in linear FE calculations is carried out for a simple structure. The steps are:

1. Define the model:
 - a. Matrix E_{DOF} – number of elements, DOF.
 - b. Element coordinates (if needed) – where is the beginning and the end of each element.
 - c. Element properties – E [Pa], A [m²], k [N/m], L [m],
2. Generate stiffness element matrices – $K^{e1}, K^{e2}, K^{e3}, \dots$
3. Assemble element matrices into the global system of equations (global stiffness matrices K).
4. Specify boundary conditions (BCs) – forces (force vector), constraints, etc.
5. Solve the global system of equations for the given BCs.
6. Evaluate element displacements, forces, etc.

Example 1:

Consider the system of three linear elastic springs, and the corresponding finite element model. The system of springs is fixed in its ends and loaded by a single load F . Evaluate element displacements and corresponding forces.



$$k = 1500 \text{ [N/m]}$$
$$F = 100 \text{ [N]}$$

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% ----- Define Topology matrix
% Edof = [ele1, dof1, dof2
%         ele2, dof1, dof2
%         ..., ..., ...]

Edof = [1  1  2
        2  2  3
        3  2  3];

k = 1500; % N/m
F = 100;  % N

% ----- Initiate stiffness matrix K and load vector f
K = zeros(3, 3);

f = zeros(3, 1);
f(2) = F;
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% ----- Generate stiffness element matrix
ep1 = k;
ep2 = 2*k;

Ke1 = spring1e(ep1)
Ke2 = spring1e(ep2)

% ----- Assemble element matrices into the global stiffness matrix K
K = assem(Edof(1,:), K, Ke2)
K = assem(Edof(2,:), K, Ke1)
K = assem(Edof(3,:), K, Ke2)

% ----- Specify boundary conditions (BCs)
% BC = [dof1    u1
%        dof2    u2
%        ...     ...]

BC = [1 0
      3 0];

% ----- Solve the global system of equations for the given BCs
[a,r] = solveq(K, f, BC)

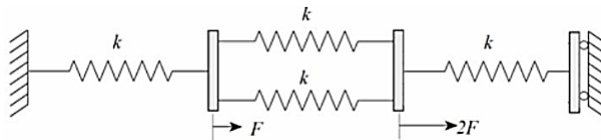
% ----- Extract element displacement
ed1 = extract_ed(Edof(1,:), a)
ed2 = extract_ed(Edof(2,:), a)
ed3 = extract_ed(Edof(3,:), a)

% ----- Extract element forces
es1 = spring1s(ep2, ed1)
es2 = spring1s(ep1, ed2)
es3 = spring1s(ep2, ed3)

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Example 2:

Consider the system of four linear elastic springs, and the corresponding finite element model. The system of springs is fixed in its ends and loaded by a single load F . Evaluate element displacements and corresponding forces.



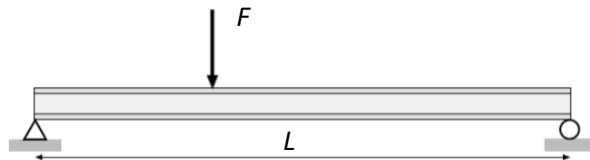
$$k = 1250 \text{ [N/m]}$$

$$F = 100 \text{ [N]}$$

Example 3:

Consider a simply supported beam. The beam is fixed in its ends and loaded by a single load F . Create a finite element model using a two dimensional beam element (**beam2e**). Discretize your model using 10 sequentially connected elements. Evaluate element displacements and corresponding forces. Plot beam displacement and estimate the change in beam deflection depending on where the force F is applied. What will happen if force F will be applied uniformly across the entire beam?

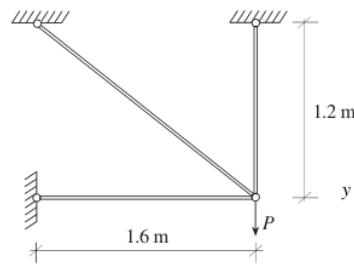
The section forces are evaluated using the function **beam2s**.



$$\begin{aligned} E &= 210 \text{ [GPa]} \\ L &= 10 \text{ [m]} \\ A &= 0.0453 \text{ [m}^2\text{]} \\ I &= 2510 \times 10^{-8} \text{ [m}^4\text{]} \\ F &= 1000 \text{ [N]} \end{aligned}$$

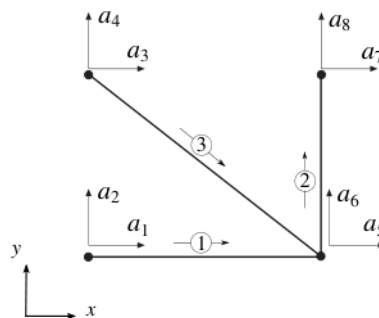
Example 4:

Consider a plane truss consisting of three bars loaded by a single force P .



$$\begin{aligned} E &= 210 \text{ GPa [N]} \\ P &= 80 \text{ [kN]} \\ A_1 &= 0.0006 \text{ [m}^2\text{]} \\ A_2 &= 0.0003 \text{ [m}^2\text{]} \\ A_3 &= 0.0010 \text{ [m}^2\text{]} \end{aligned}$$

The corresponding finite element model consists of three elements and eight degrees of freedom.



The element stiffness matrices Ke_1 , Ke_2 , and Ke_3 are computed using **bar2e**. The section forces are calculated using **bar2s** from element displacements.

Example 5:

Consider a plane truss, loaded by a single force P . The corresponding finite element model in (a) consists of seventeen elements and eighteen degrees of freedom. A similar model in (b) has fifteen elements and sixteen degrees of freedom. Compare the results of models (a) and (b).

The element matrices K_e are computed by the function **bar2e**. Normal forces are evaluated using the function **bar2s**.

$E = 210 \text{ GPa [N]}$

$A = 0.0025 \text{ [m}^2\text{]}$

$P = 0.5 \text{ [MN]}$

