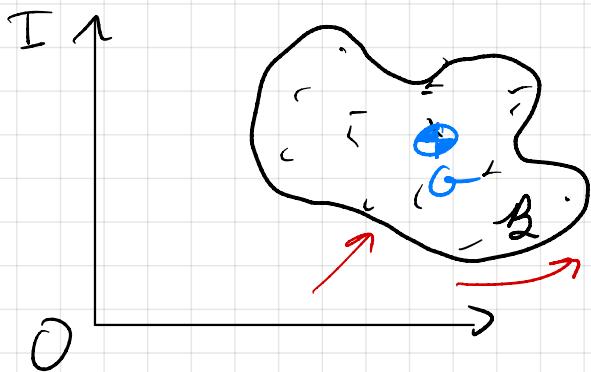


Chapter 9

10/24/24





a rigid body is a MPS

$N \rightarrow \infty$ "continuum"

body is free to translate & rotate
(in the plane)

Recall
Ch 6

$$\bar{F}^{(ext)} = m_G \bar{a}_{G/O} \quad (\text{G behaves like a particle})$$

$$\boxed{\bar{F}_G = m_G \bar{a}_{G/O}}$$

Euler's First Law

* governs translational motion
of the COM of a RB

Recall
Ch 6

$$\bar{F}_{G/O} = \frac{1}{m_G} \sum_{k=1}^N m_k \bar{F}_{k/O}$$

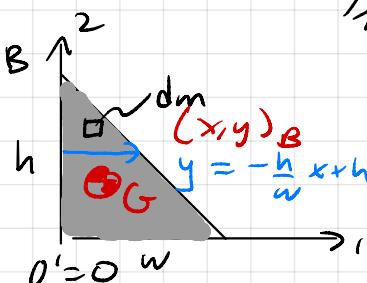
$$\boxed{\bar{F}_{G/O} = \frac{1}{m_G} \int_B dm \bar{F}_{dm/O}} \quad (N \rightarrow \infty)$$

$$2D : \frac{\text{mass}}{\text{area}} = \text{density} \quad \frac{dm}{dA} = \rho (\bar{F}_{dm/O})$$

$$3D : \frac{\text{mass}}{\text{vol}} = \text{density} \quad \frac{dm}{dV} = \rho (\bar{F}_{dm/O})$$

$$\Rightarrow \bar{F}_{G/O} = \frac{1}{m_G} \int_B \rho dA \bar{F}_{dm/O} \quad (\text{in 2D})$$

Ex 9.1

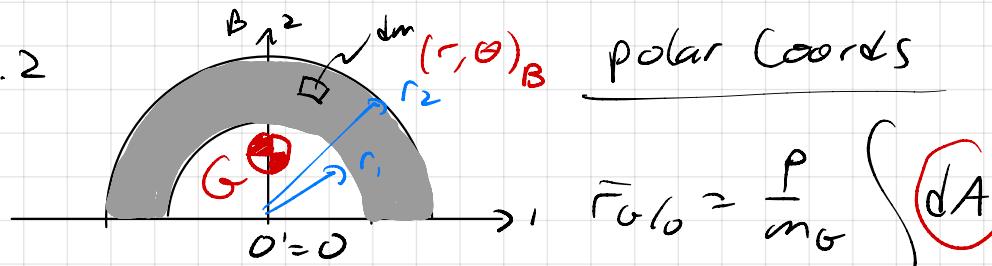


$$\frac{1}{2}wh = \frac{m}{\rho}$$

Cartesian Coordinates

$$\begin{aligned} \bar{F}_{G/O} &= \frac{1}{m} \int \int dx dy (x \hat{b}_1 + y \hat{b}_2) \\ &= \frac{1}{3} (w \hat{b}_1 + h \hat{b}_2) \end{aligned}$$

Ex 9.2



$$\bar{r}_{G/B} = \frac{\rho}{m_G} \quad \boxed{dA} \bar{r}_{dm/B}$$

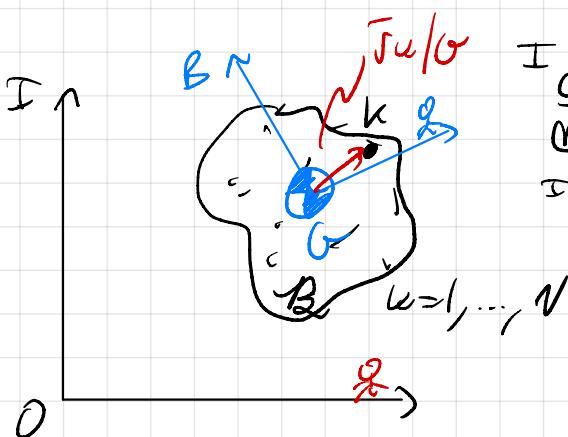
$$dA = r dr d\theta$$

$$\begin{aligned}\bar{r}_{G/B} &= \frac{\rho}{m} \int_{r_1}^{r_2} \int_0^{\pi} r dr d\theta (r \cos \theta \hat{b}_1 + r \sin \theta \hat{b}_2) \\ &= \frac{4(r_2^3 - r_1^3)}{3\pi(r_2^2 - r_1^2)} \hat{b}_2 \quad \frac{\pi r_2^2 - \pi r_1^2}{2} = \frac{\rho}{m}\end{aligned}$$

Rotational dynamics

Recall Ch. 7

$$\overset{\text{I}}{\bar{h}_0} = \overset{\text{I}}{\bar{h}_{B/B}} + \overset{\text{I}}{\bar{h}_G} \quad \text{separation of AM}$$



$$\begin{aligned}\overset{\text{I}}{\frac{d}{dt}}(\overset{\text{I}}{\bar{h}_0}) &= \overset{\text{I}}{\bar{M}_G}^{(\text{ext})} = \sum_{k=1}^N \overset{\text{I}}{\bar{M}_{k/B}}^{(\text{ext})} \\ \overset{\text{I}}{\bar{h}_G} &= \sum_{k=1}^N \overset{\text{I}}{\bar{h}_{k/B}} \\ &= \sum_{k=1}^N \bar{r}_{k/B} \times m_k \overset{\text{I}}{\bar{v}_{k/G}}\end{aligned}$$

Recall Ch. 8

$$\overset{\text{I}}{\frac{d}{dt}}(\bullet) = \overset{\text{B}}{\frac{d}{dt}}(\bullet) + \overset{\text{I}}{\bar{w}} \overset{\text{B}}{\times} (\bullet)$$

$$\overset{\text{I}}{\bar{v}_{k/G}} = \overset{\text{I}}{\frac{d}{dt}}(\bar{r}_{k/G}) \stackrel{\text{TE}}{=} \overset{\text{B}}{\frac{d}{dt}}(\bar{r}_{k/G}) + \overset{\text{I}}{\bar{w}} \overset{\text{B}}{\times} \bar{r}_{k/G} \quad \text{transport equation}$$

$$\Rightarrow \text{For a rigid body } \overset{\text{I}}{\bar{h}_G} = \sum_{k=1}^N \bar{r}_{k/G} \times m_k (\overset{\text{I}}{\bar{w}} \overset{\text{B}}{\times} \bar{r}_{k/B})$$

App B p. 640 vector triple product identity

$$\bar{a} \times (\bar{b} \times \bar{c}) = \bar{b}(\bar{c} \cdot \bar{a}) - \bar{c}(\bar{a} \cdot \bar{b})$$

$$\Rightarrow {}^I\bar{h}_\omega = \sum_{k=1}^N m_k \left[{}^I\bar{w}^B \left(\bar{r}_{k/G} \cdot \bar{r}_{k/G} \right) - \bar{r}_{k/G} \left(\bar{r}_{k/G} \cdot {}^I\bar{w}^B \right) \right]$$

$$= \underbrace{\left(\sum_{k=1}^N m_k \|\bar{r}_{k/G}\|^2 \right)}_{} {}^I\bar{w}^B$$

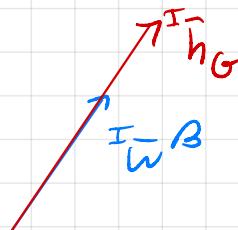
$$= O(20)$$

II_G Moment of inertia

$$\Rightarrow {}^I\bar{h}_\omega = II_G {}^I\bar{w}^B \quad \text{where } II_G = \sum_{k=1}^N m_k \|\bar{r}_{k/G}\|^2$$

* AM is parallel to AV (2D or 3D)

* MOI is scalar (2D or 3D)



Rotation
Dynamics

$$\frac{d}{dt}({}^I\bar{h}_\omega) = \bar{M}_\omega$$

* Euler's law

For a continuum RB ($N \rightarrow \infty$)

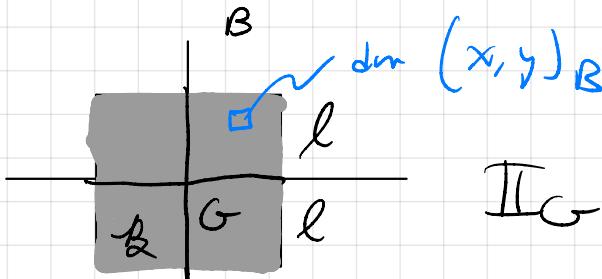
$$II_G = \int_B dm \|\bar{r}_{dm/G}\|^2$$

$$dm = \rho dA \quad (2D)$$

$$dm = \rho dV \quad (3D)$$

Ex 9.1

MOI
(Cartesian
coords.)



$$\rho = \frac{m}{4l^2}$$

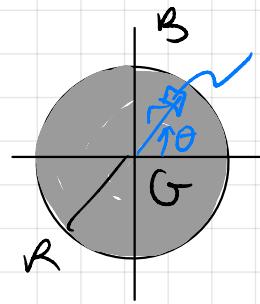
$$II_G = \rho \int_{-l}^{l} \int_{-l}^{l} dx dy (x^2 + y^2)$$

$$= \dots = 2ml^2/3$$

Ex 9.8

MOI

(polar
coords)



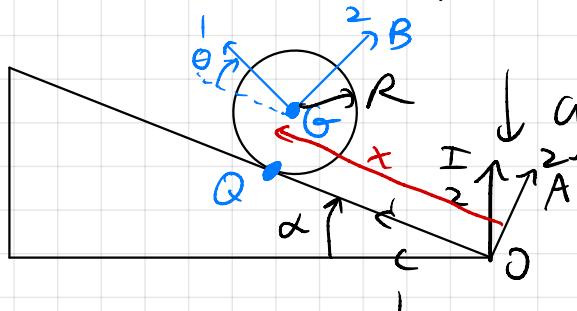
$$\rho = \frac{m}{\pi R^2}$$

$$dm(r, \theta)_B$$

$$\begin{aligned} I_G &= \rho \int_0^{2\pi} \int_0^R r^3 r dr d\theta \\ &= \dots = \frac{1}{2} m R^2 \end{aligned}$$

Ex 9.9 Disc rolling w/o slipping ($\ddot{x} = -R\ddot{\theta}$) constraint

	\hat{b}_1	\hat{b}_2
\hat{e}_1	$C\theta$	$-S\theta$
\hat{e}_2	$S\theta$	$C\theta$
\hat{e}_3	\hat{b}_1	\hat{b}_2
\hat{b}_3	Cx	$-Sx$
\hat{e}_3	Sx	Cx



$$I = (0, \hat{e}_1, \hat{e}_2, \hat{e}_3) \text{ inertial}$$

$$A = (0, \hat{a}_1, \hat{a}_2, \hat{a}_3) \text{ ramp}$$

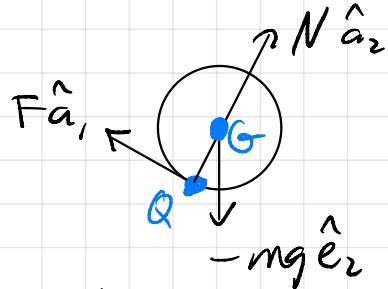
$$B = (G, \hat{b}_1, \hat{b}_2, \hat{b}_3) \text{ body}$$

$$M = 6N - K = 6 - 2 - 2 = 2 \text{ DOF}$$

translational rotation constraints

Euler 1st Law: translation

$$\bar{F}_G = m_G \bar{a}_G / \bar{t}_0$$



$$\bar{r}_{G/G} = x \hat{a}_1 + R \hat{a}_2$$

$$\bar{I} \bar{a}_{G/G} = \ddot{x} \hat{a}_1$$

$$\bar{I} \bar{a}_{G/G} = \ddot{x} \hat{a}_1$$

$$\bar{F}_{a1} + N_{a2} - mg \hat{e}_2 = m \ddot{x} \hat{a}_1$$

$$F - mg Sx = m \ddot{x}$$

in known

Euler 2nd Law: Rotation

$$\frac{d}{dt} (\bar{I} \bar{h}_G) = \bar{M}_G$$

$$\bar{M}_G = \sum_{k=1}^N \bar{M}_k / G$$

$$= \sum_{k=1}^N \bar{r}_{k/G} \times \bar{F}_k$$

$$= \bar{r}_{Q/G} \times \bar{N} + \bar{r}_{Q/G} \times \bar{F} + \bar{r}_{G/G} \times \bar{W} = 0$$

$$= -R \hat{a}_2 \times \bar{F}_{a1} = R \bar{F} \hat{a}_3$$

$$\bar{I} \bar{h}_G = \bar{I}_G \bar{w}^B = \bar{I}_G \dot{\theta} \hat{b}_3$$

$$\bar{I}_G \ddot{\theta} \hat{b}_3 = R \bar{F} \hat{a}_3 \Rightarrow \bar{I}_G \ddot{\theta} = RF$$

2 unknowns

$$\textcircled{1} \quad \ddot{x} = -R\ddot{\theta}$$

constraint law/ls
rot. dyn.

$$\textcircled{2} \quad I_G\ddot{\theta} = RF$$

trans. dyn

$$\textcircled{3} \quad F - mg\sin\alpha = m\ddot{x}$$

$$\frac{I_G\ddot{\theta}}{R} - mg\sin\alpha = m\ddot{x}$$

$$\frac{I_G}{R}\frac{\ddot{x}}{-R} - mg\sin\alpha = m\ddot{x} \Rightarrow \left(m + \frac{I_G}{R^2}\right)\ddot{x} = -mg\sin\alpha$$

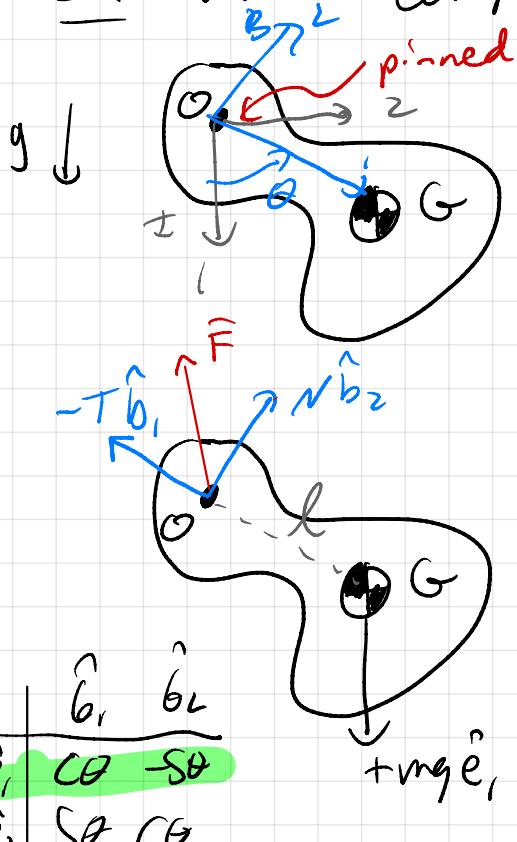
$$\ddot{x} = \frac{-mg\sin\alpha}{m + I_G/R^2}$$

$$I_G = k_2 m R^2$$

$$\Rightarrow \ddot{x} = -\frac{2}{3}g\sin\alpha$$

$$\ddot{\theta} = \frac{2}{3}\frac{g\sin\alpha}{R}$$

Ex 9.15 Compound Pendulum



$$I_{\bar{h}_\theta} = I_G \bar{w} \bar{b} = I_G \dot{\theta} \bar{b}_3$$

$$\bar{I} \frac{d}{dt} (\bar{I}_{\bar{h}_\theta}) = \bar{M}_\theta$$

$$\bar{M}_\theta = \bar{r}_{G/\theta} \times \bar{W} + \bar{r}_{O/\theta} \times (\bar{T} + \bar{N})$$

$$= -l \dot{\theta} \times N \hat{b}_2 = -l N \hat{b}_3$$

$$I_G \ddot{\theta} = -l N$$

2 unknowns

Option #1 use Euler's 1st Law

Option #2 use parallel axis theorem

	\hat{b}_1	\hat{b}_2
\hat{e}_1	$C\theta - S\theta$	
\hat{e}_2	$S\theta C\theta$	

* key idea: for a pinned RB, write the dynamics relative to the pin and use the parallel axis theorem.

let Q be a point fixed in B and I

$$\boxed{\frac{d}{dt} (\overset{I}{h}_Q) = \overset{I}{M}_Q} \quad \text{pinned rigid body}$$

$$\overset{I}{h}_Q = \overset{I}{I}_Q \overset{I}{\omega}^B$$

$$\boxed{\overset{I}{I}_Q = \overset{I}{I}_G + m_Q || \overset{I}{r}_{Q/G} ||^2} \quad \text{parallel axis theorem}$$

$$\overset{I}{M}_Q = \sum_{k=1}^N M_{k/Q} = \sum_{k=1}^N \overset{I}{r}_{k/Q} \times \overset{I}{F}_k$$

Ex Compound Pendulum let Q = O

$$\frac{d}{dt} (\overset{I}{h}_Q) = \overset{I}{M}_Q$$

$$\overset{I}{h}_Q = (\overset{I}{I}_G + m l^2) \dot{\theta} \hat{b}_3$$

$$\overset{I}{M}_Q = \overset{I}{r}_{G/O} \times \overset{I}{W}$$

$$= l \hat{b}_1 \times mg \hat{e}_1 = l \hat{b}_1 \times mg (\cos \hat{b}_2, -\sin \hat{b}_2)$$

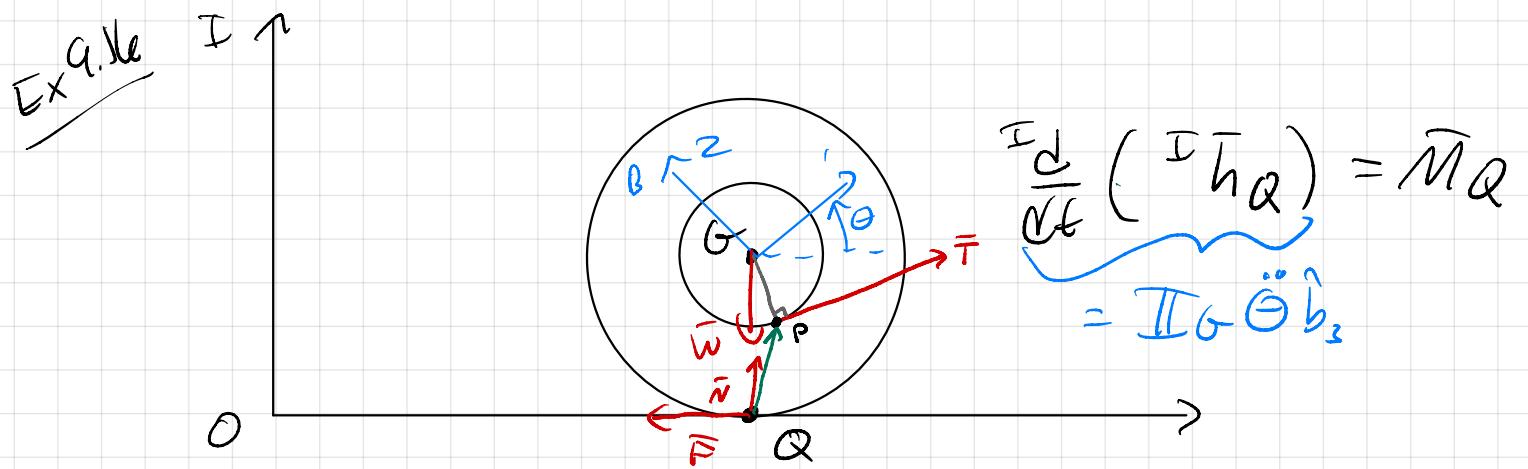
$$= -mgl \sin \hat{b}_3$$

$$(\overset{I}{I}_G + m l^2) \ddot{\theta} \hat{b}_3 = -mgl \sin \hat{b}_3$$

$$\boxed{\ddot{\theta} = \frac{-mgl \sin \theta}{\overset{I}{I}_G + m l^2}}$$

Comparison Simple Pendulum $\overset{I}{I}_G \rightarrow 0$

$$\ddot{\theta} = -\frac{g}{l} \sin \theta$$

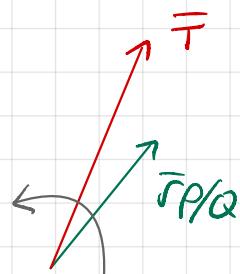


$$\bar{M}_Q = \bar{r}_{P/Q} \times \bar{T}$$



\bar{M}_Q into the page

\Rightarrow wheel rolls right



\bar{M}_Q out of the page

\Rightarrow wheel rolls left

Work & Energy Rigid Body

Recall $T_D = T_{G/O} + T_G$ Separation of KE

$$T_{G/O} = \frac{1}{2} \sum_{i=1}^N m_i \parallel \bar{v}_{i/O} \parallel^2 \text{ translational KE}$$

$$T_G = \frac{1}{2} \sum_{i=1}^N m_i \parallel \bar{v}_{i/G} \parallel^2$$

$$\bar{I} \bar{v}_{i/G} = \frac{d}{dt} (\bar{r}_{i/G}) = \underbrace{\frac{d}{dt} (\bar{r}_{i/G})}_{=0} + \bar{\omega} \times \bar{r}_{i/G}$$

$$T_G = \frac{1}{2} \sum_{i=1}^N m_i (\bar{\omega}^B \times \bar{r}_{i/G}) \cdot (\bar{\omega}^B \times \bar{r}_{i/G})$$

scalar triple product

$$\bar{a} \cdot (\bar{b} \times \bar{c}) = (\bar{a} \times \bar{b}) \cdot \bar{c}$$

$$T_G = \frac{1}{2} \sum_{i=1}^N m_i \bar{\omega}^B \cdot \bar{r}_{i/G} \times \bar{\omega}^B \times \bar{r}_{i/G}$$

$$= \frac{1}{2} \bar{\omega}^B \cdot \sum_{\alpha=1}^N m_\alpha \bar{r}_\alpha/G \times \bar{\omega}^B \times \bar{r}_\alpha/G$$

$= I_G \bar{\omega}^B$

$$= \frac{1}{2} \bar{\omega}^B \cdot I_G \bar{\omega}^B = \frac{1}{2} \bar{\omega}^B \cdot I_G$$

$T_G = \frac{1}{2} I_G ||\bar{\omega}^B||^2$
rotational KE

$$W = W_{G/0} + W_G$$
separation of work

$$W_{G/0} = \int_{t_0} T_G \cdot \bar{v}_{G/0} dt = \int_{t_0} \bar{F}_G \cdot \bar{v}_{G/0} dt$$

$$W_G = \int \bar{M}_G \cdot \bar{\omega}^B dt$$

WE #1 $T_G(t_2) = T_G(t_1) + W_{G/0} + W_G$

translation $T_{G/0}(t_2) = T_{G/0}(t_1) + W_{G/0}$

rotation $T_G(t_2) = T_G(t_1) + W_G$

WE #2 $U_G(t_2) = U_G(t_1) - W^{(c)}$

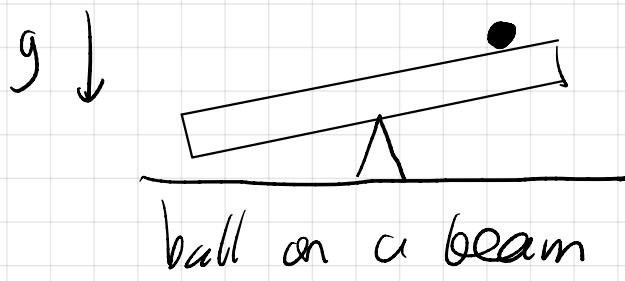
WE #3 $E_G(t_2) = E_G(t_1) + W^{(nc)}$

$T_{G/0}(t_2) + T_G(t_2) + U_G(t_2) =$

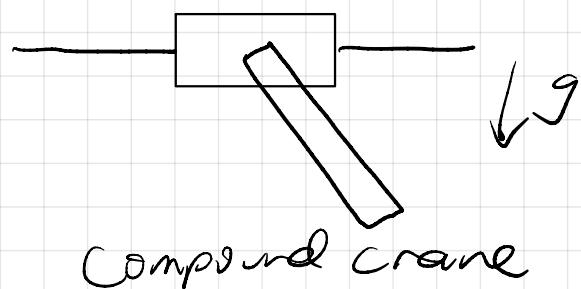
$T_{G/0}(t_1) + T_G(t_1) + U_G(t_1) + W_{G/0}^{(nc)} + W_G^{(nc)}$

E_G is conserved if $W^{(nc)} = W_{G/0}^{(nc)} + W_G^{(nc)} = 0$

A Collection of RBQ Particles



ball on a beam



Compound crane