Counting Encurchments

- => Count the number of complete loops the diagram makes around -1.
- =) A <u>Clockwise</u> loop counts as +1 encirclement A <u>Counter-clockwise</u> loop counts as -1 encirclement
- => Diagrams may have both CW or CCW loops around -1
- => Let Ncw(L) be the Net Number of CW encirclements for Nyquist diagram of L (i.e. result of adding contribution of each loop Using the ±1 convention above).

Easy Way to Count Encirclements

"Ray trick"

- => Draw a ray radially outward from I in any direction
- => Looking along the ray, away from -I
 - => Count +I each time diagram crosses
 ray from left to right.
 - => Count -1 each time ray is crossed right to left.
- => Same result regardless of ray direction
- => Choose direction with least number of intersections for easiest counting.

Example:
$$L(s) = \frac{K_B}{(r's+1)^3}$$
 $N_{CW} = \emptyset$
 $V_{CW} = \emptyset$
 V_{CW

Example:
$$L(s) = \frac{K_B}{(r's+1)^3}$$
 $N_C \omega = (+1) + 1$
 $= 2$
 $\omega \to \infty$
 Re

A more complicated Example:

$$L(s) = \frac{K_8(T_1S+1)^2}{(T_2S+1)^3} T_2 >> T_1 > \emptyset$$

$$1 \text{ Im}$$

$$N_{CW} = + 2 \text{ if -1 here}$$

$$R_e$$

$$N_{CW} = 0 \text{ if -1 here}$$

A more complicated Example:

Nyquist Stability Theorem

For an arbitrary transfer function G(s), define

Nyquist showed:

$$N_{c\omega}(L) = P_{R}(T) - P_{R}(L)$$

Re-arranging:
$$P_{R}(T) = P_{R}(L) + N_{cw}(L)$$
Want to predict — Known

Note: PR(T)≥Ø always. If you compute PR(T)<Ø

Implication

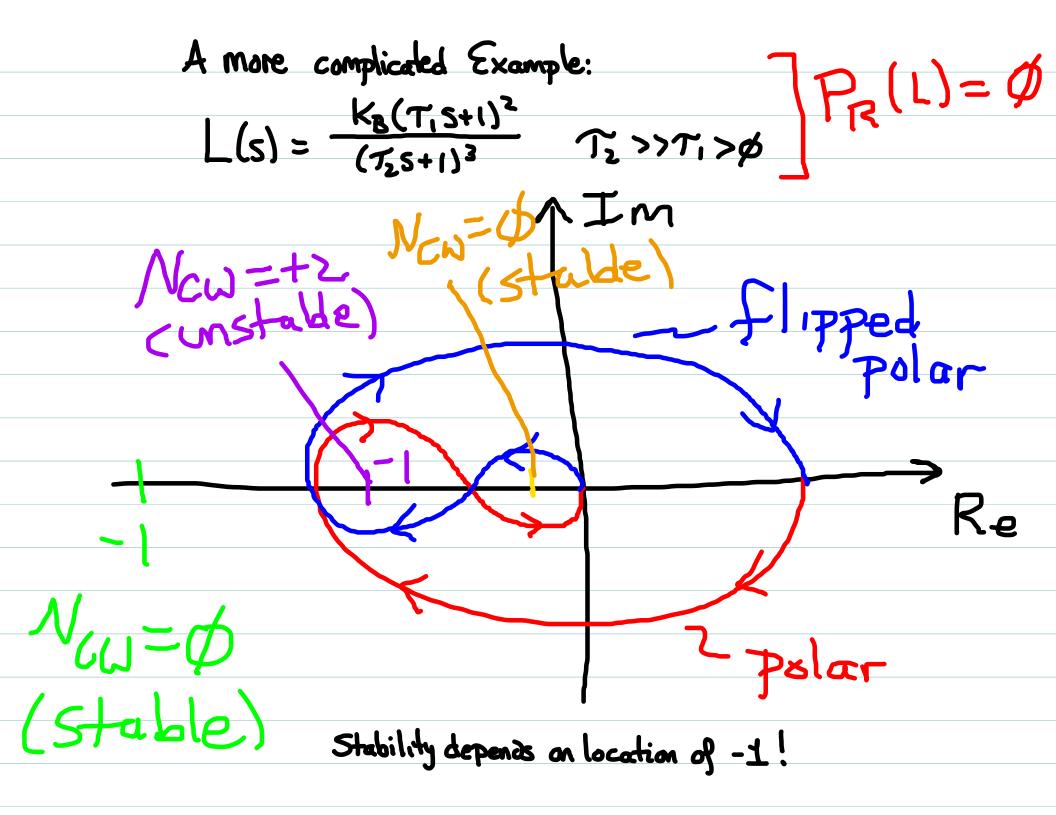
=> We must have
$$P_R(T)=\emptyset$$
 (Stable closed-loop system)

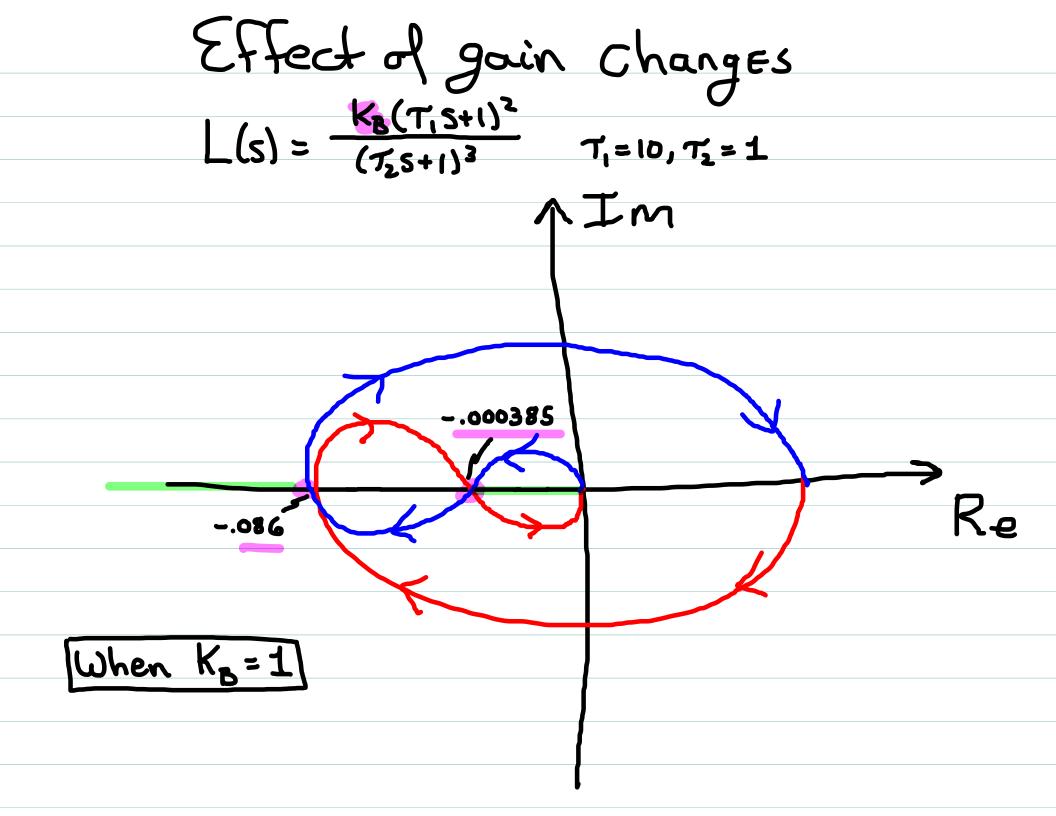
$$\Rightarrow N_{c\omega}(L) = -P_{R}(L)$$
 (Stability condition)

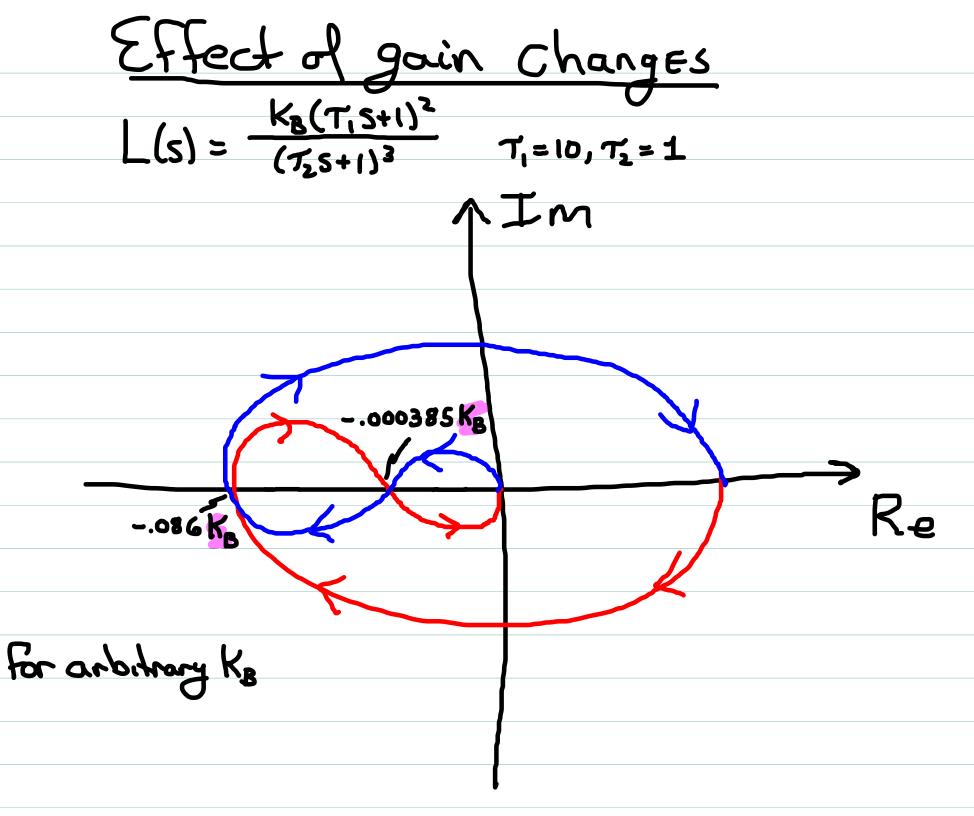
i.e. Nyguist diagram must show a net negative number of encirclements, equal to number of unstable poles of L(s).

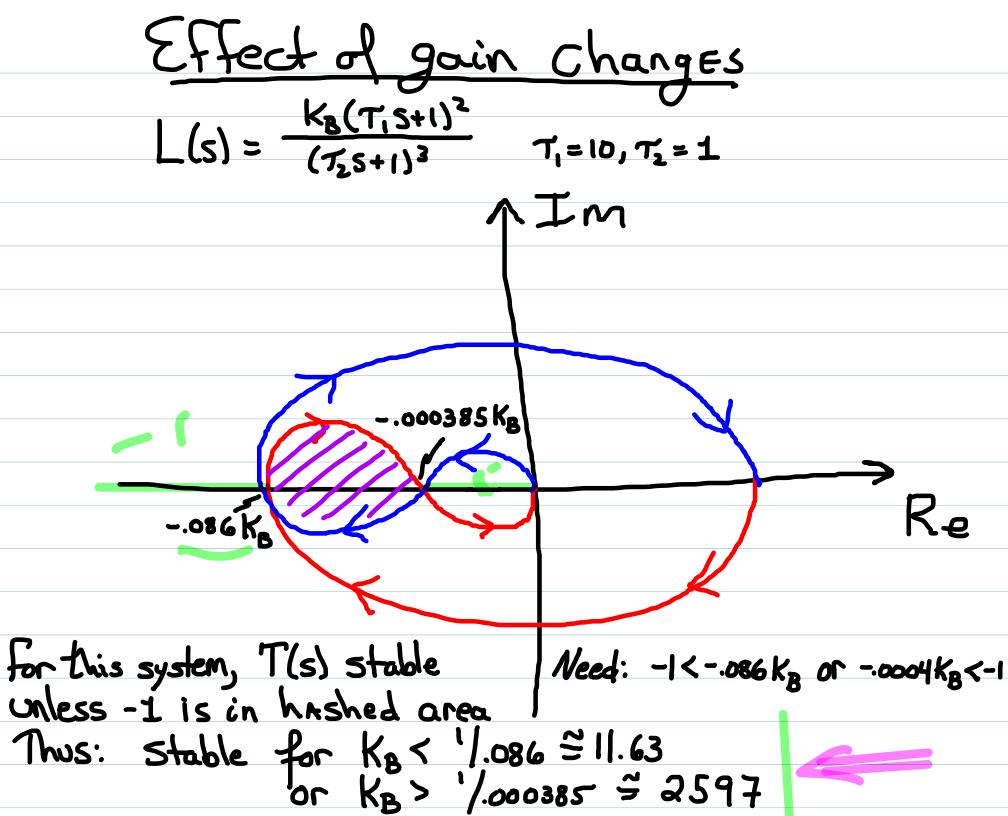
Recall negative CW encirclements are CCW encirclements

Note: if $P_R(L) = \emptyset$ (L(s) is stable) then the diagram must show NO (Ø) Net encirclement









Note: Gain change is easy to accomplish with compensator:

H(s) = K (=> u(t) = Ke(t) "proportional" control)

L(s) = H(s)G(s) = KG(s) here

 \Rightarrow $(K_B)_L = K(K_B)_6$

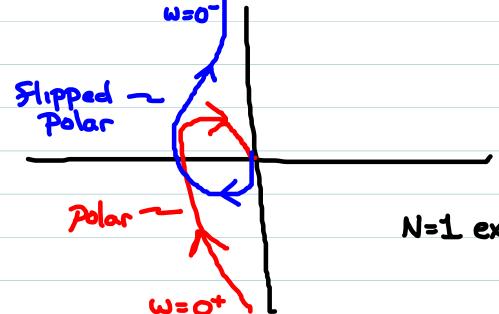
However, gain change only affects "size" of polar (hence location of -1 relative to loops in Nyquist).

More substantial changes to polar/Nyquist diagram (changes to number and/or location of cts/oops)
require also zeros/poles in H(s).

Nyquist Diagram for N>Ø Systems

When L(s) has type N>Ø (one or more poles at origin) the first step to creating Nyquist diagram is same:

However, the resulting diagram is Not connected; both halfs have "tails" parallel to coordinate AXES



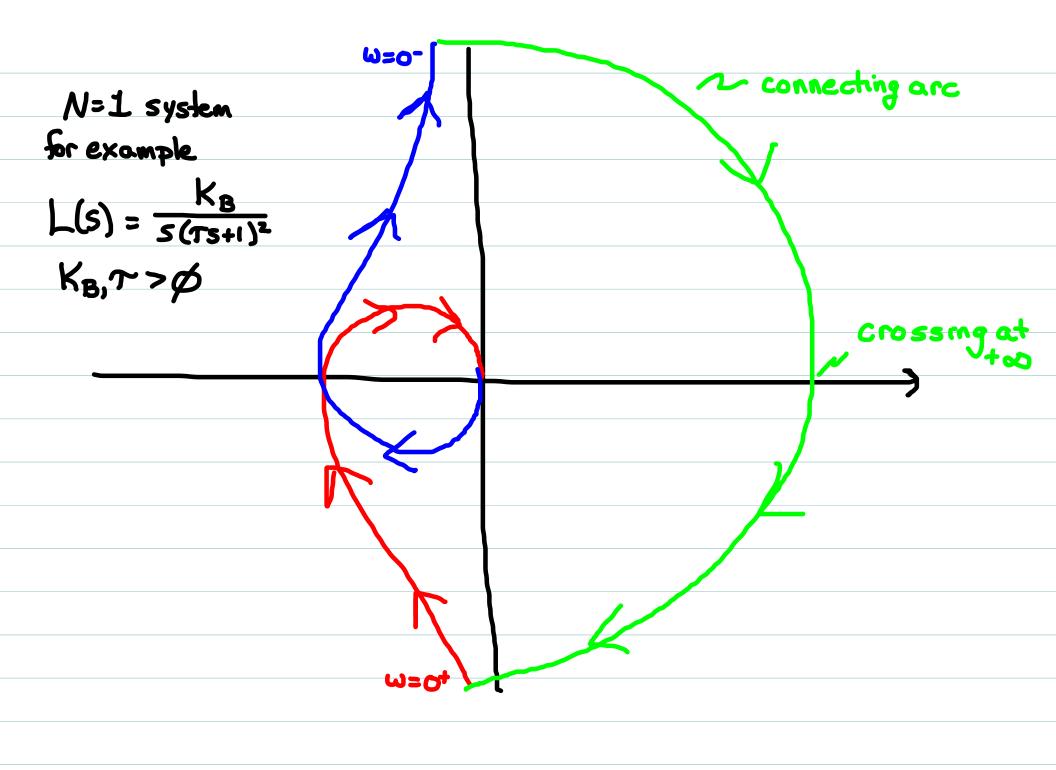
Completing the diagram, N>Ø

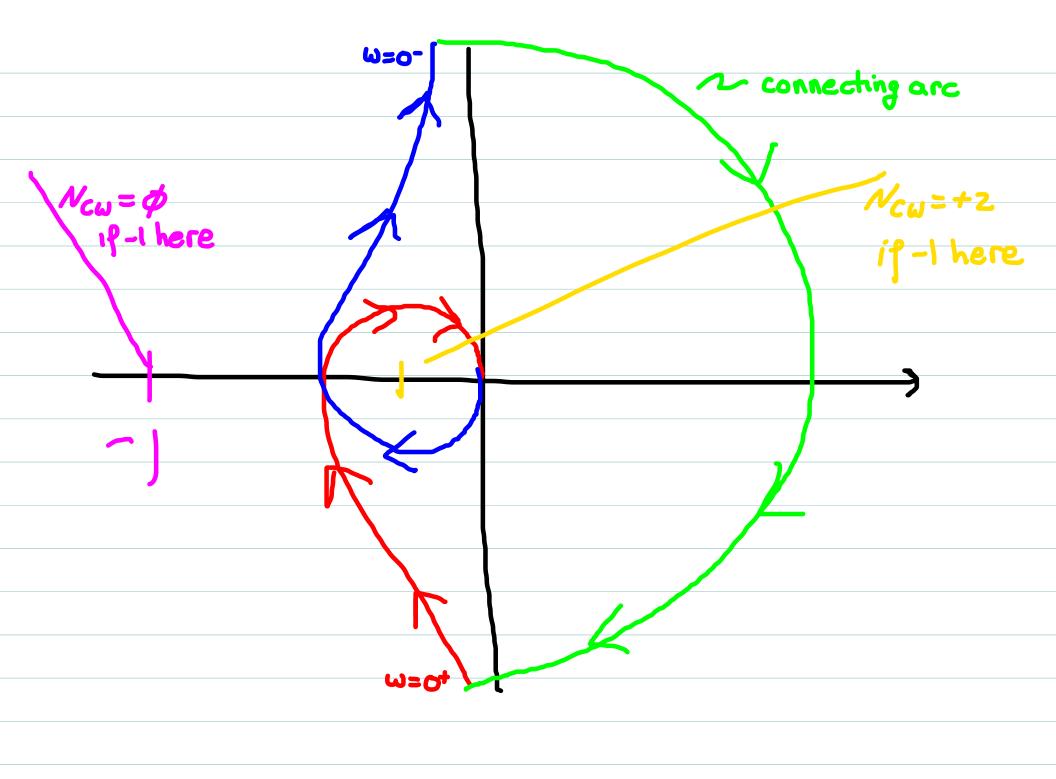
=> Connect the w=0 tail of flipped polar to with a clockwise circular arc of total rotation NT

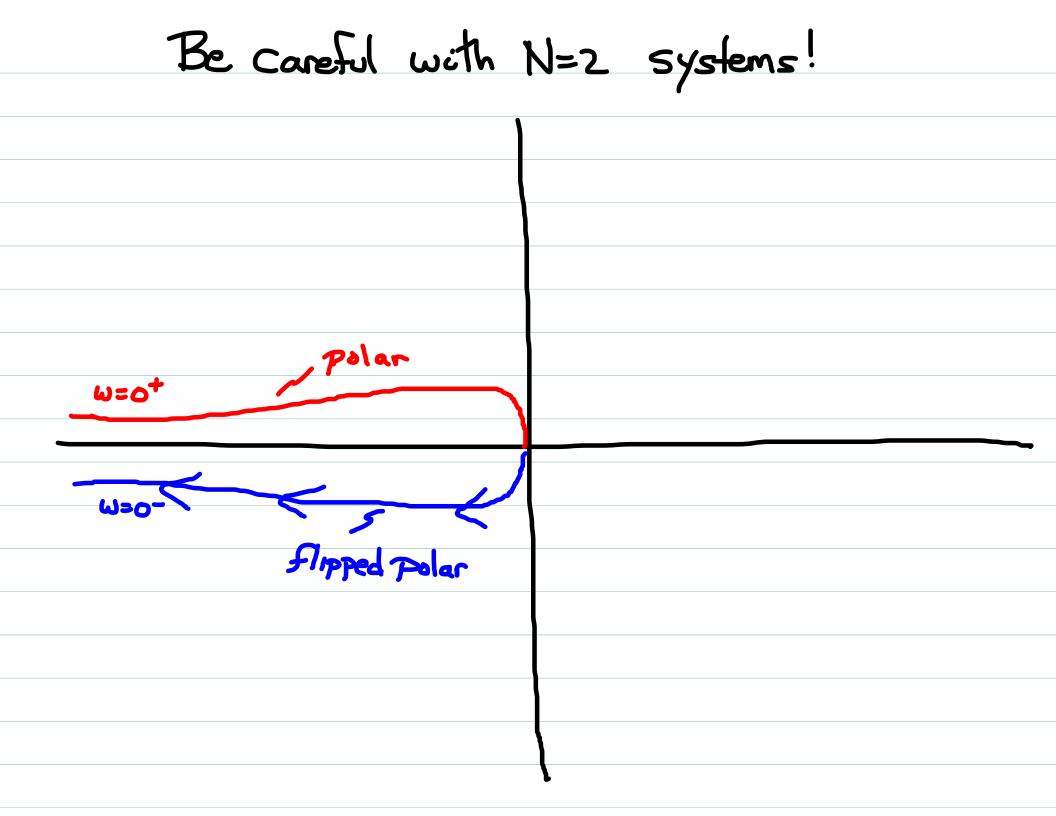
(i.e. 1/2 circle for every pole at origin in L(s))

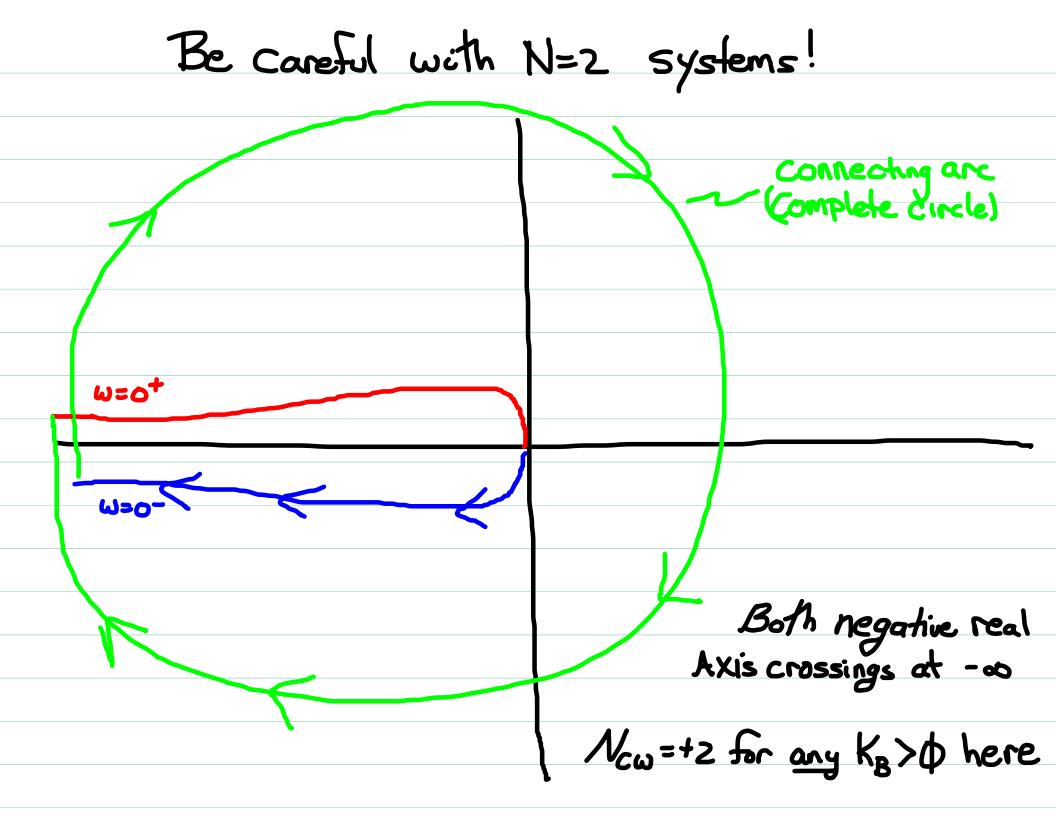
Note: Connecting arc has infinite radius, although we draw it as finite.

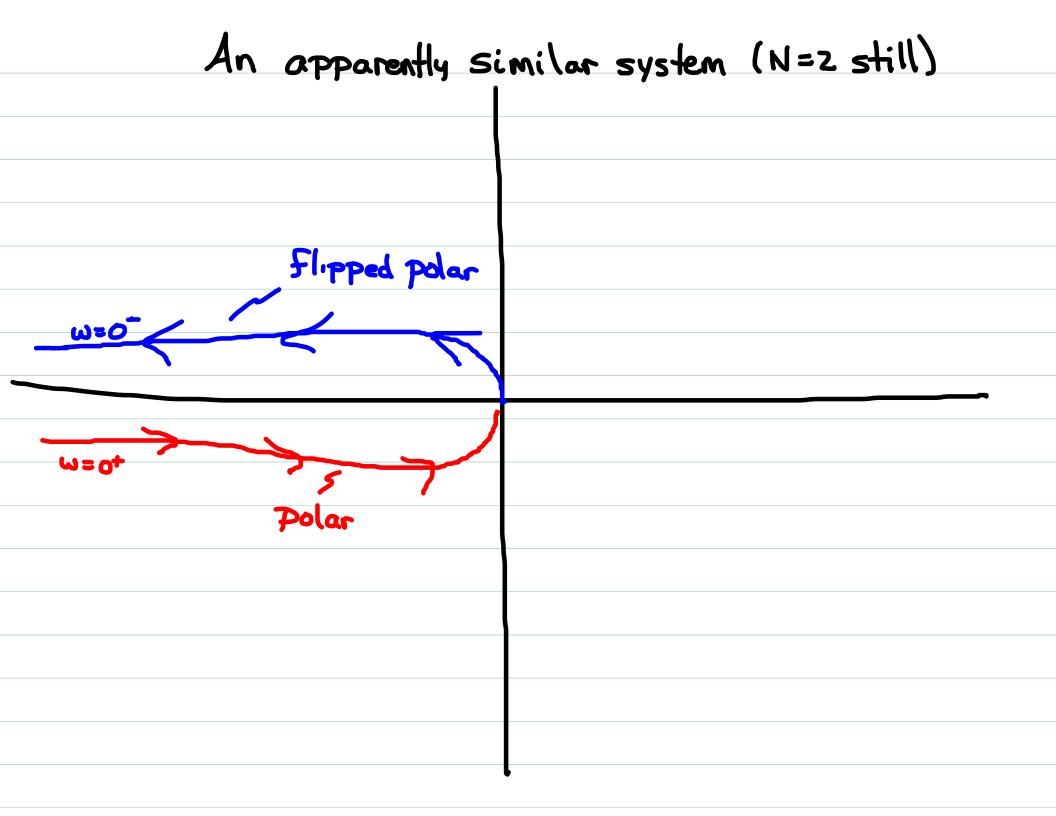
=> After connecting tails, compute Now(L) as before.

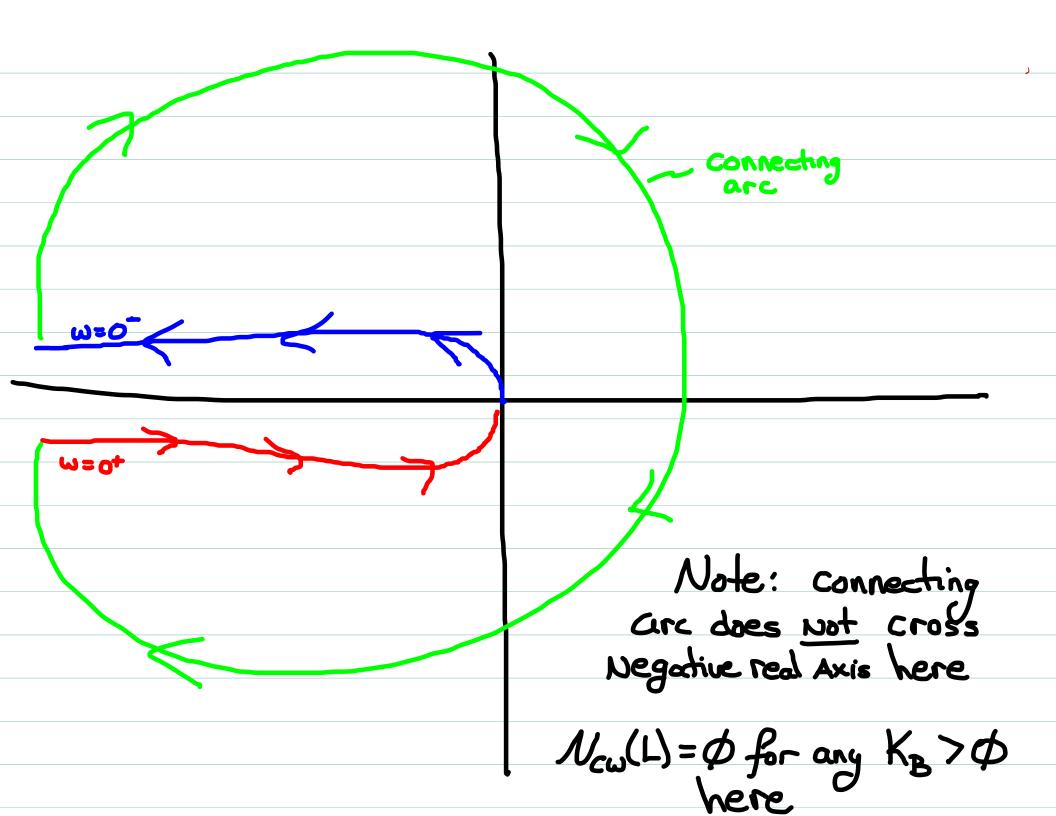












Utility of gair/phase margin

- => a, 8 measure how close polar comes to -1
 - => If design is Nominally stable (Nyquist shows required number of encurdements of -1), then
 - A, 8 measure how much Nyquist can Change in a pure gain or phase fashion, before -I would enter a different loop, changing the number of encirclements.
 - Thus: a, & are measures of the "tolerance" of the system's stability to gain/phase changes in L(s).
 - => Relative stability measures.

Robustness (classical)

As measures of the tolerance of the control system Stability to changes in shape of Nyquist, gain and phase margin are measures of the robustness of the design.

That is, the ability of the design to tolerate moved errors which would create pure gain or pure phase errors in L(s)

Typically caused by errors in mobel of G(s), since

$$L(s) = G(s)H(s)$$

and there is no uncertainty in H(s).

Classical Robustness Requirements

A "robustly stable" design thus requires:

=> Correct number of Nyquist encirclements

(AND) => Large 1a1, 181

Typical professional requirements

 \Rightarrow $|a_{dB}| > 6$ (i.e. a > 6dB or a < -6dB)

=> | X | > 30°

Requirement on a is physically equivalent to no more than a factor of Z uncertainty on gain of G(s)