

T&D Mini-Series – Session 8, Wrap Up

2/28/2023



Prepared By: Brandon M. Foster, P.E.



Agenda

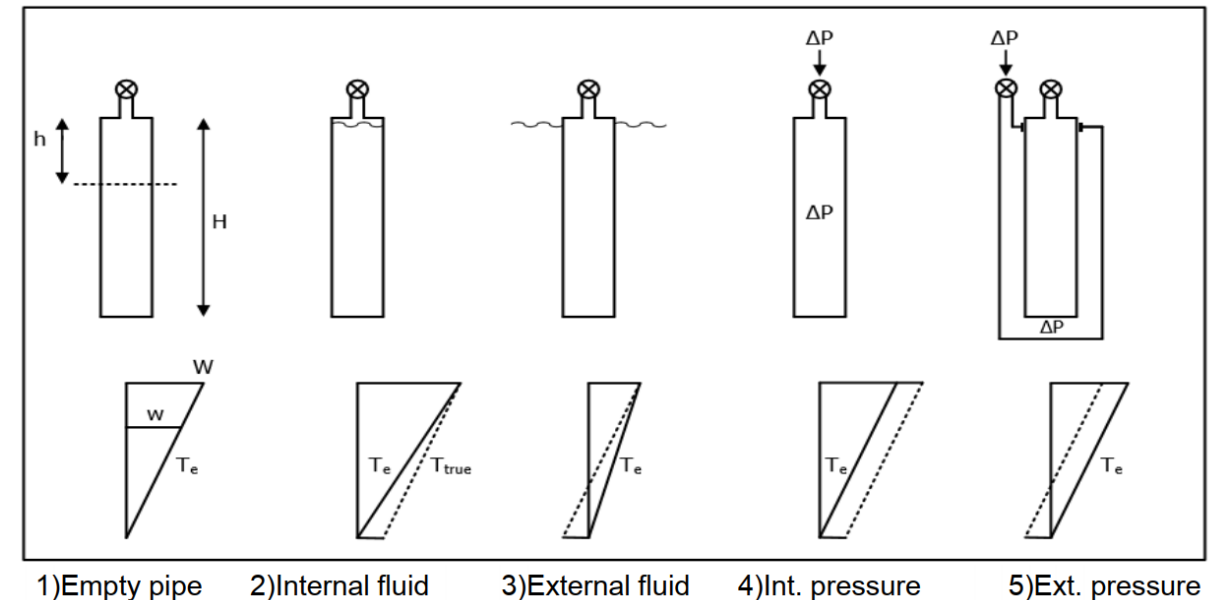
- Recap
- Real vs. Effective Tension
- How to determine max tension and torque (practical exercise)
- Q&A

T&D Mini-Series Recap

- Session 1 – [Brandon Foster](#) – Intro: [Slides](#) [Video](#)
- Session 2 – [Truls Larsen](#) – Ancillary Applications: [Slides](#) [Video](#)
- Session 3 – [Dr. Catalin Teodoriu](#) – Basic Physics: [Slides](#) [Video](#)
- Session 4 – [Dr. Stephane Menand](#) – Soft vs. Stiff String: [Slides](#) [Video](#)
- Session 5 – [Neil Armstrong](#) – Model Techniques and Applications: [Slides](#) [Video](#)
- Session 6 – [Mitch Abahusayn](#) – Special Operations: [Slides](#) [Video](#)
- Session 7 – [Devi Subramaniam](#) – Realtime Operations: [Slides](#) [Video](#)

“What is “True” vs. Effective Tension?”

- $T_e = T_{true} - P_i A_i + P_e A_e$
- T_e = Effective tension, lbs
- T_{true} = True tension, lbs
- P_i = Internal pressure, psi
- A_i = Internal cross-sectional area, in²
- P_e = External pressure, psi
- A_e = External cross-sectional area, in²



SPE-174785

- Effective tension is what matters when we are calculating T&D and evaluating buckling
- “True” tension is what we would read on a strain gauge. That does not mean the effective tension isn’t “real”.
- “True” tension is the starting point of the calculation and then we take internal and external pressure into account

“What Is the Maximum Allowable Tension / Torque?”

- Who? – Who are we talking to? Engineer, Company Man, Driller, Grandma?
- What? – Limited by what? The drill pipe, BHA, top drive, or *current emotional state*?
- When? – Near TD, at 16:00 on Friday, in the middle of the night, or after we got stuck?
- Where? – Assuming we have a weak point in the string, where is it currently located?
- Why? – Why have we accepted our current limits as statement of fact?
- *HOW* do we figure all this out?

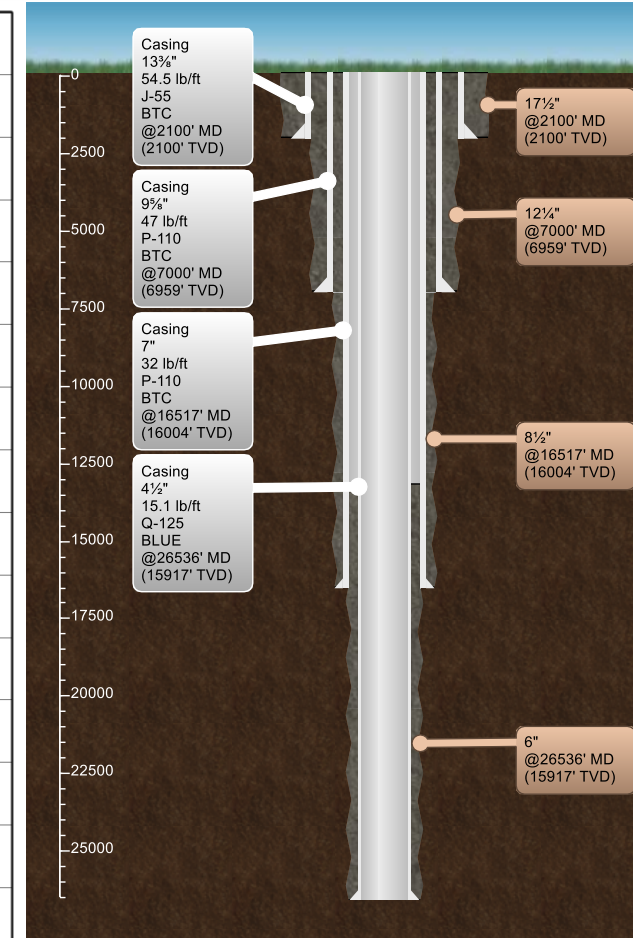
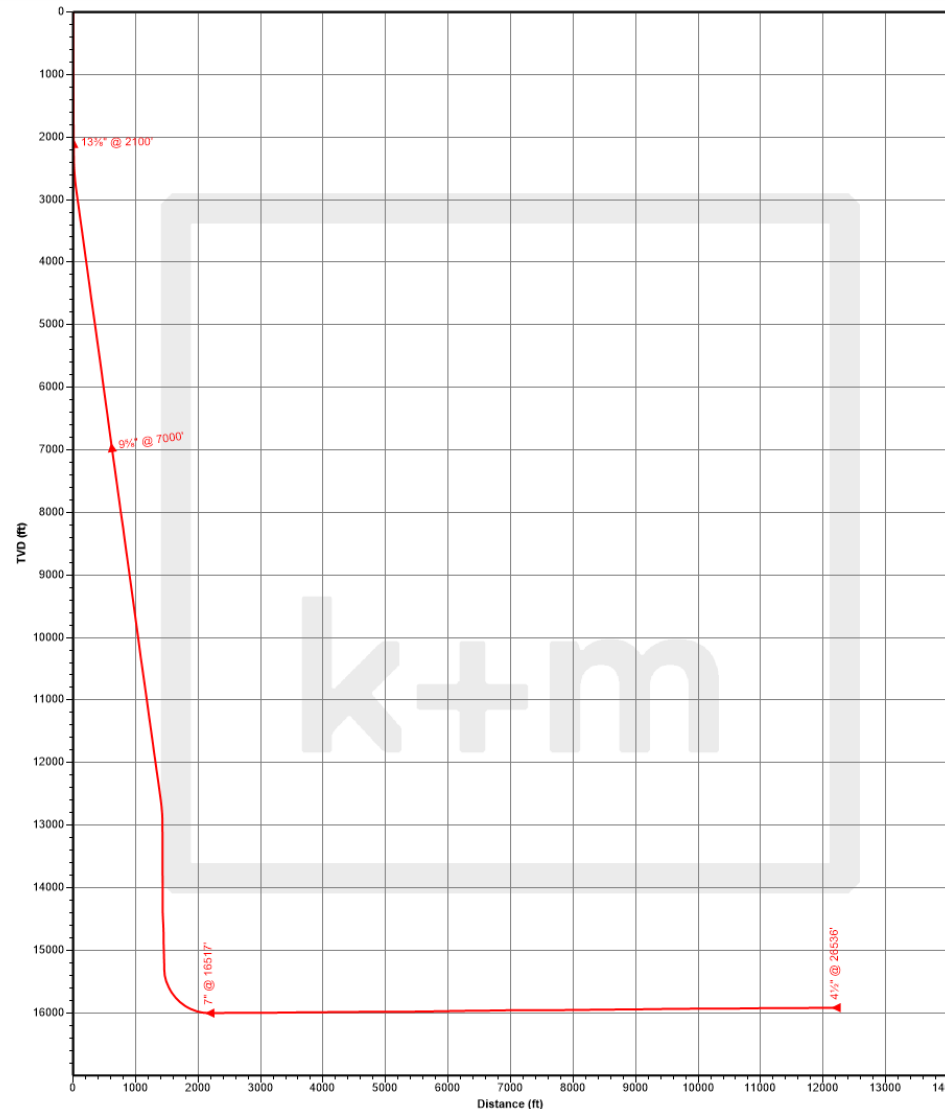


Wellpath

- Deep TVD – 16,000'
- 1,500' Stepout
- 2-Mile Lateral
- 1.5°/100' Nudge
- 8°/100' BUR
- 4-String Design
 - Casing points based on offset vertical wells
- Sandstone target interval
 - Hard, abrasive, 25 drilling days
- 13,000 psi multistage frac completion

“Just like a Bakken well”

“XYZ company is drilling 3-mile laterals – we can do this, no problem”



Drill Pipe Spec

- We are renting 4" 14# S135 XT39
- Spec Sheet says for Premium (80% remaining body wall thickness):
 - Tube Tension = 403,500 lbs
 - Tube Torque = 32,800 ft-lbs
 - Max makeup = 22,300 ft-lbs
- We need to apply a Design Factor to ensure that we don't exceed ratings and allow for some wear while drilling.
- Typically we use 80% / 1.25 DF for the tube
 - Tube Tension = 322,800 lbs
 - Tube Torque = 26,240 ft-lbs
- Some Operators also limit torque to 90% of MUT. Not uncommon to go to 100%
 - Max Torque = 20,000 ft-lbs

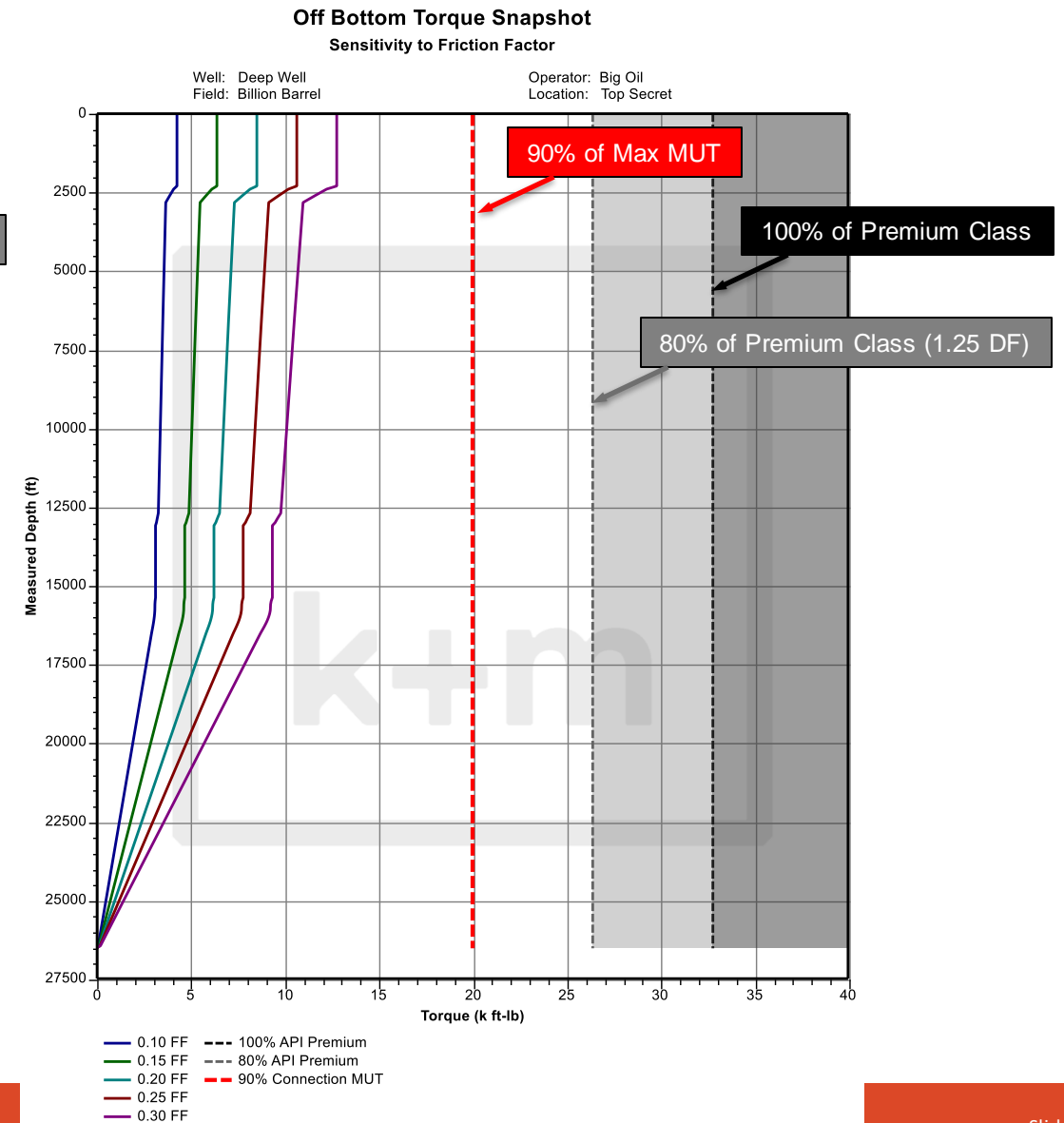
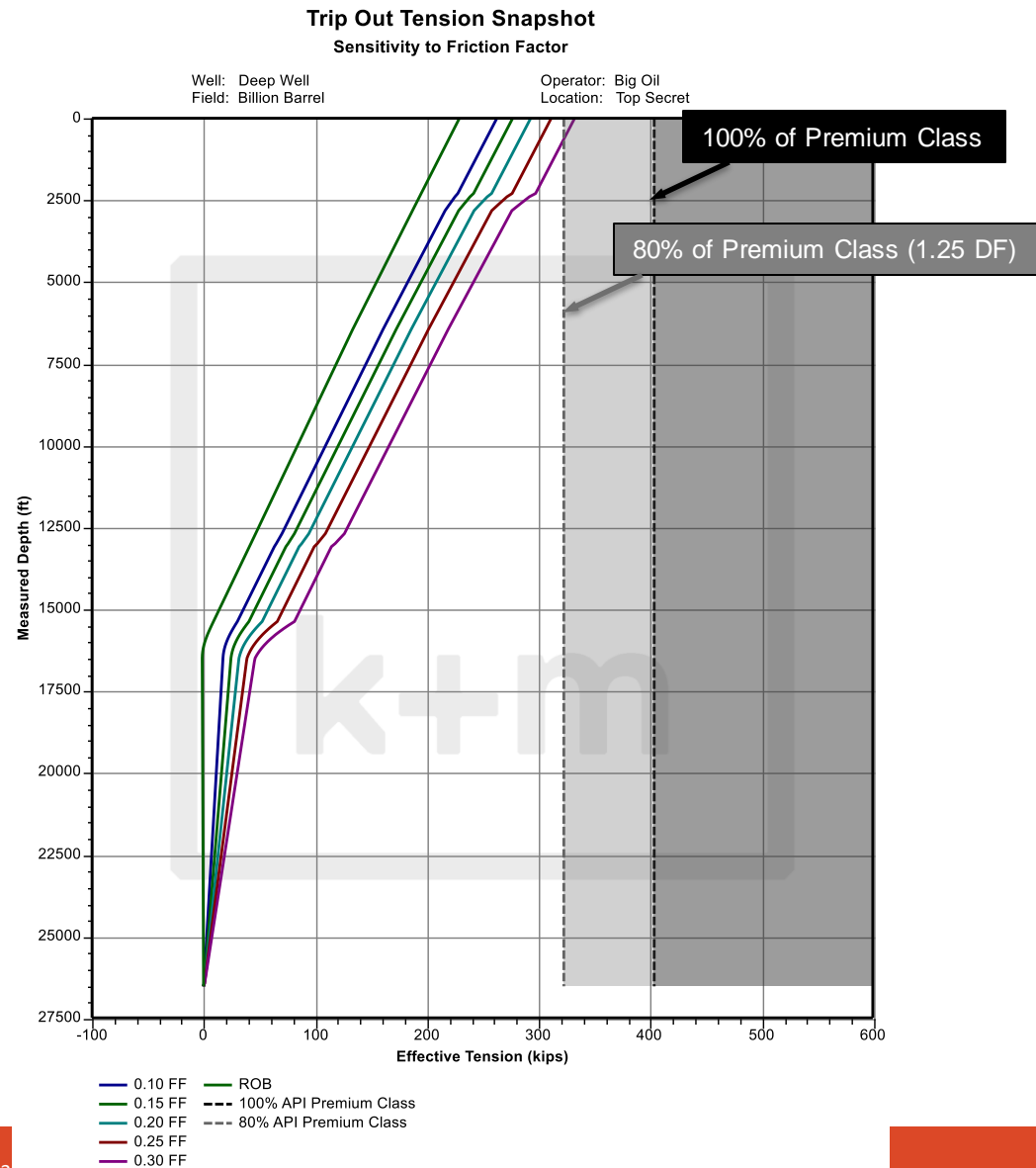
FOR REFERENCE ONLY



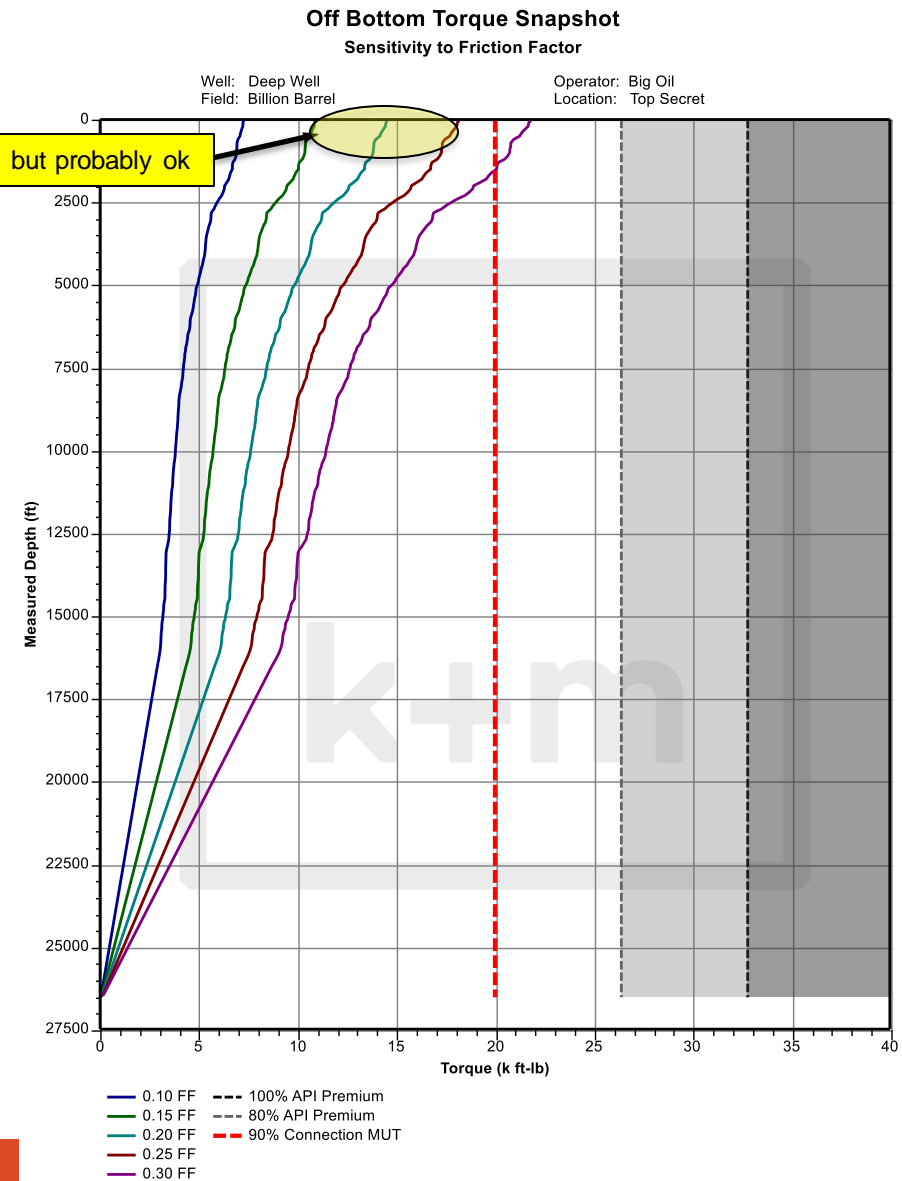
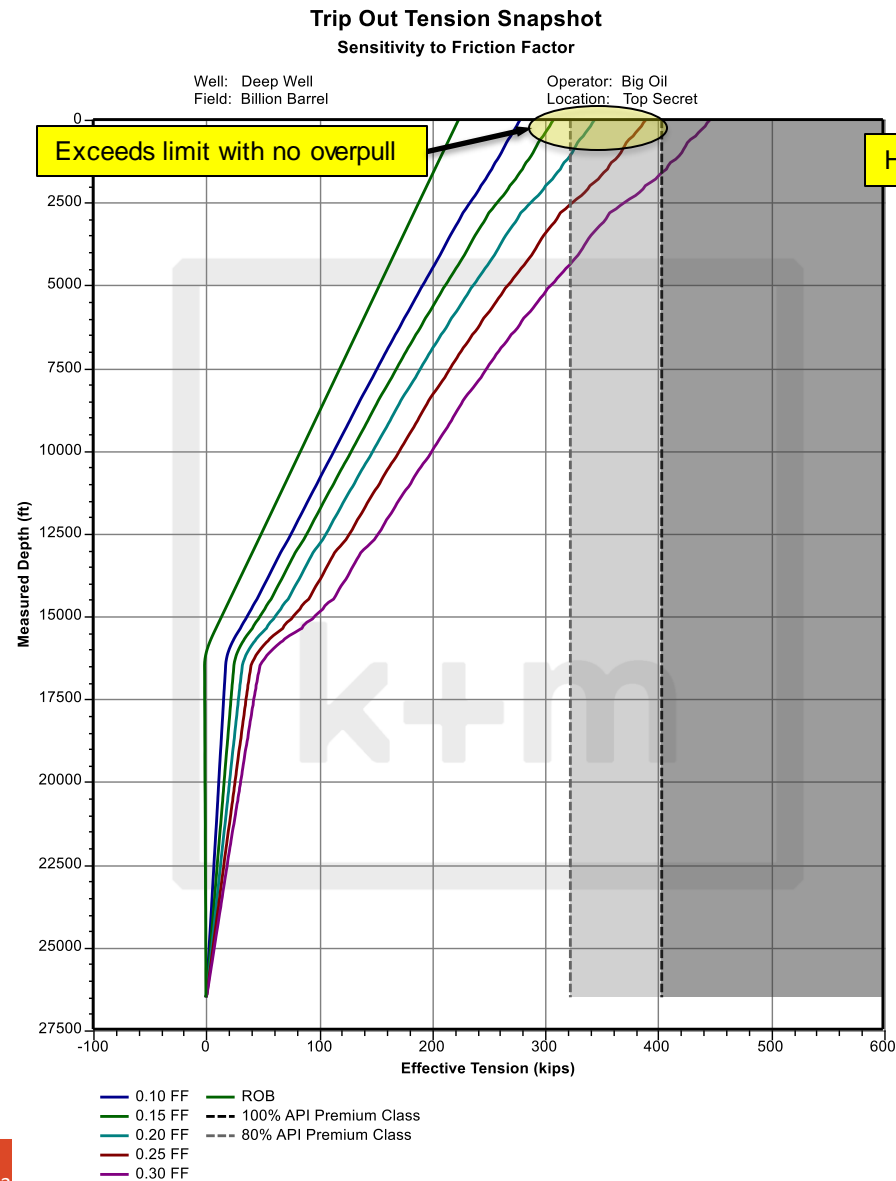
Drill Pipe Performance Characteristics ©				Pipe Size and Weight: 4" 14.00 IU Pipe Grade: S-135 Range: 3 Tool Joint: 4.875 X 2.563 XT39		
Pipe						
		New	Premium		New	Premium
OD (in)	4.000	3.868		Torsional strength (ft-lbs)	41,900	32,800
Wall thickness (in)	0.330	0.264		Tensile strength (lbs)	513,600	403,500
ID (in)	3.340	3.340				
Calculated plain end weight (lbs/ft)	12.921	10.151		80% Torsional strength (ft-lbs)	33,500	26,200
Note: Premium properties are calculated based on uniform OD and wall thickness.						
Cross sectional area pipe body (in²)	3.805	2.989		Pressure capacity (psi)	19,491	18,428
Cross sectional area OD (in²)	12.566	11.751		Collapse capacity (psi)	20,141	13,836
Cross sectional area ID (in²)	8.762	8.762				
Section modulus (in³)	3.229	2.523				
Polar section modulus (in³)	6.458	5.046				
Tool Joint (120000psi material Yield Strength)				Drill pipe assembly with Grant Prideco XT39 eXtreme™ Torque Connection		
XT39	OD (in)	4.875				
	ID (in)	2.563				
	Pin tong length (in)	10.0		Adjusted weight (lbs/ft)	14.91	
	Box tong length (in)	15.0		Approximate length (ft)	43.71	
	Torsional Strength (ft-lbs)	37,100		Fluid displacement (gal/ft)	0.228	
	Max Recommended Make-up Torque (ft-lbs)	22,300		Fluid capacity (gal/ft)	0.446	
	Min Recommended Make-up Torque (ft-lbs)	12,500		Drift size (in)	2.438	
	Balance OD (in)	4.992				
	Tensile strength (lbs)	729,700				
	Tool joint/Drill pipe torsional ratio (New pipe)	0.89				
	Tool joint/Drill pipe torsional ratio (Prem pipe)	1.13				
	Min OD for premium class (in)	4.531				
Minimum make-up is based on shoulder separation caused by bending.						

Brian Christen 4/22/2002

4" 14# Drill Pipe, No Tortuosity, Uniaxial Limits



4" 14# Drill Pipe, Tortuosity, Uniaxial Limits



Combined Loading (aka, VME or Triaxial Loading)

$$VME = \sqrt{\sigma_r^2 + \sigma_\theta^2 + \sigma_z^2 - \sigma_r\sigma_\theta - \sigma_r\sigma_z - \sigma_\theta\sigma_z + 3\tau^2}$$

- Converts stress in multiple directions to one “equivalent” direction
- The equivalent stress is then compared to yield strength of material
- *Can also re-arrange the equation to solve for tension, torque, etc.*

σ_r = Radial stress

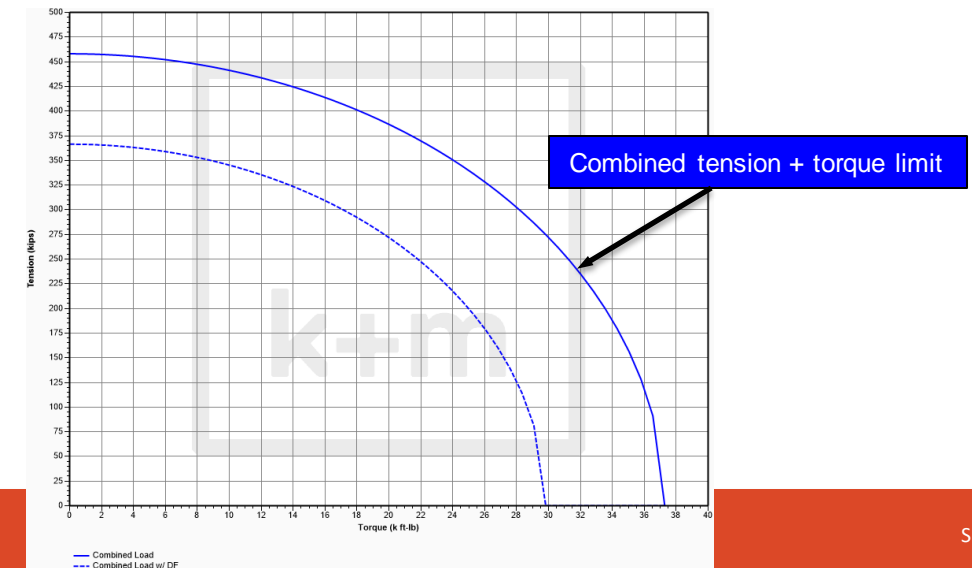
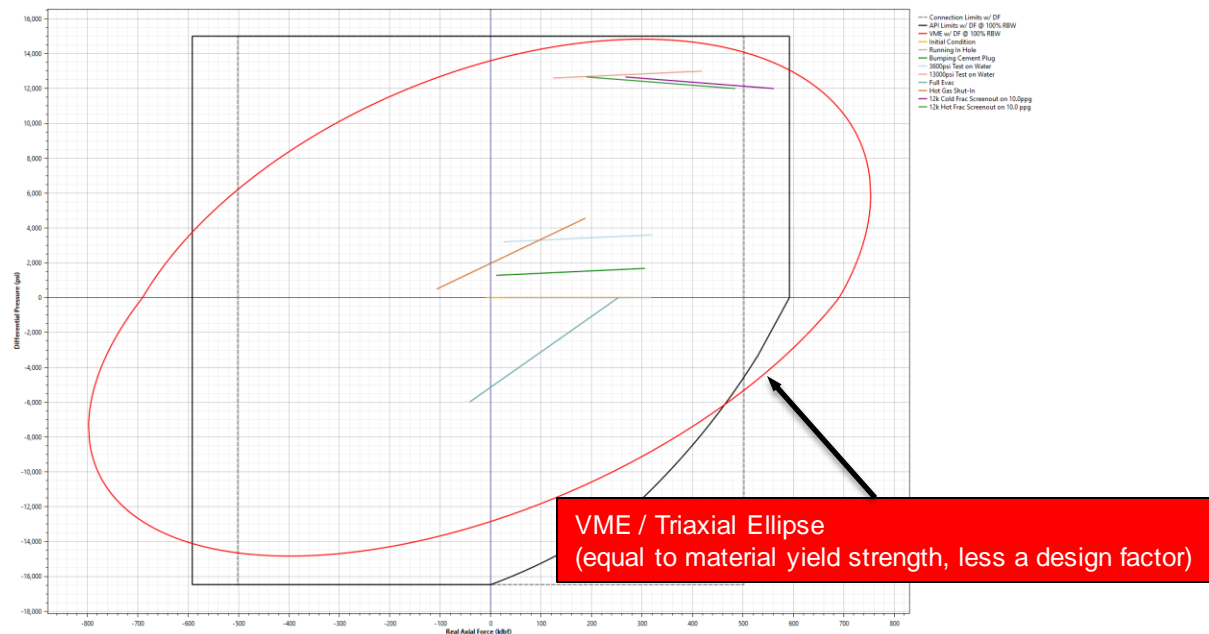
σ_θ = Hoop stress

σ_z = Effective axial stress ($\sigma_a + \sigma_b$)

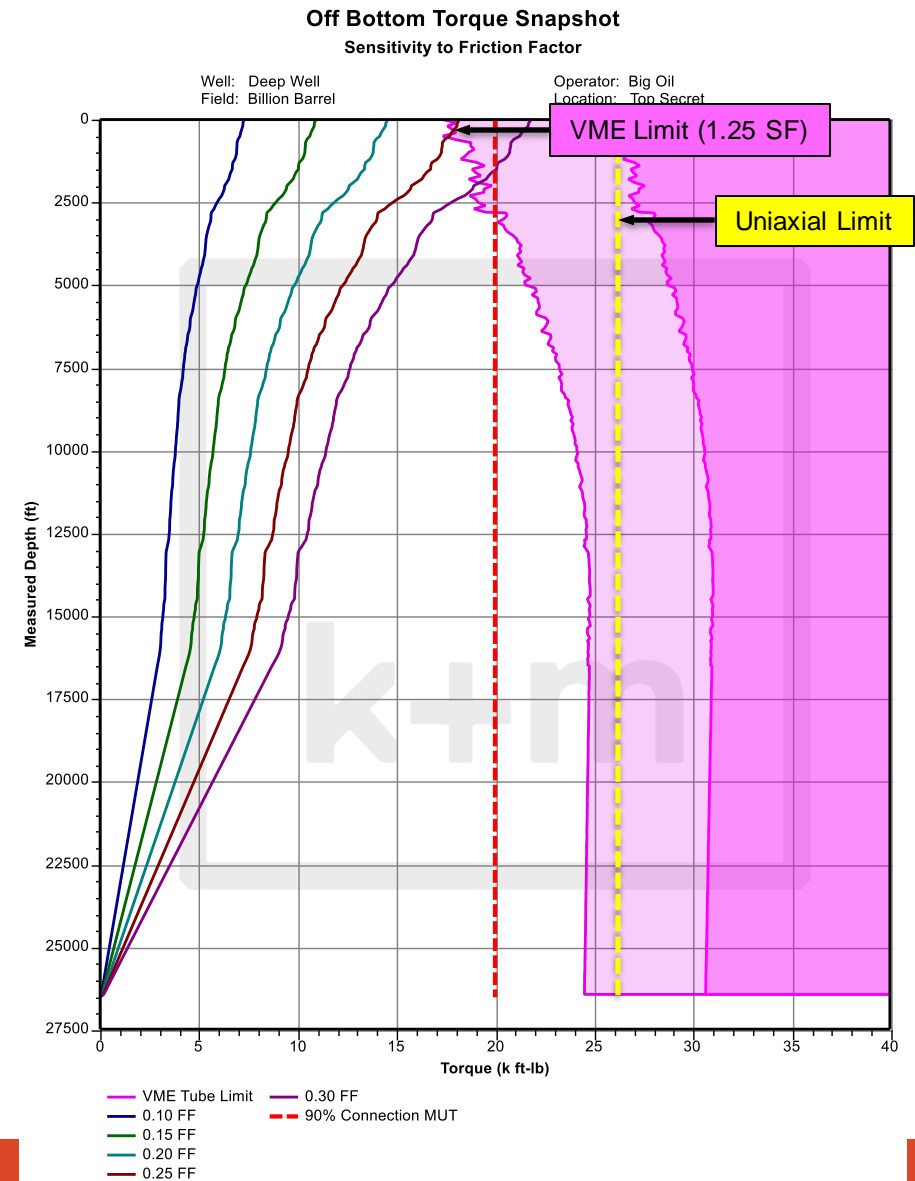
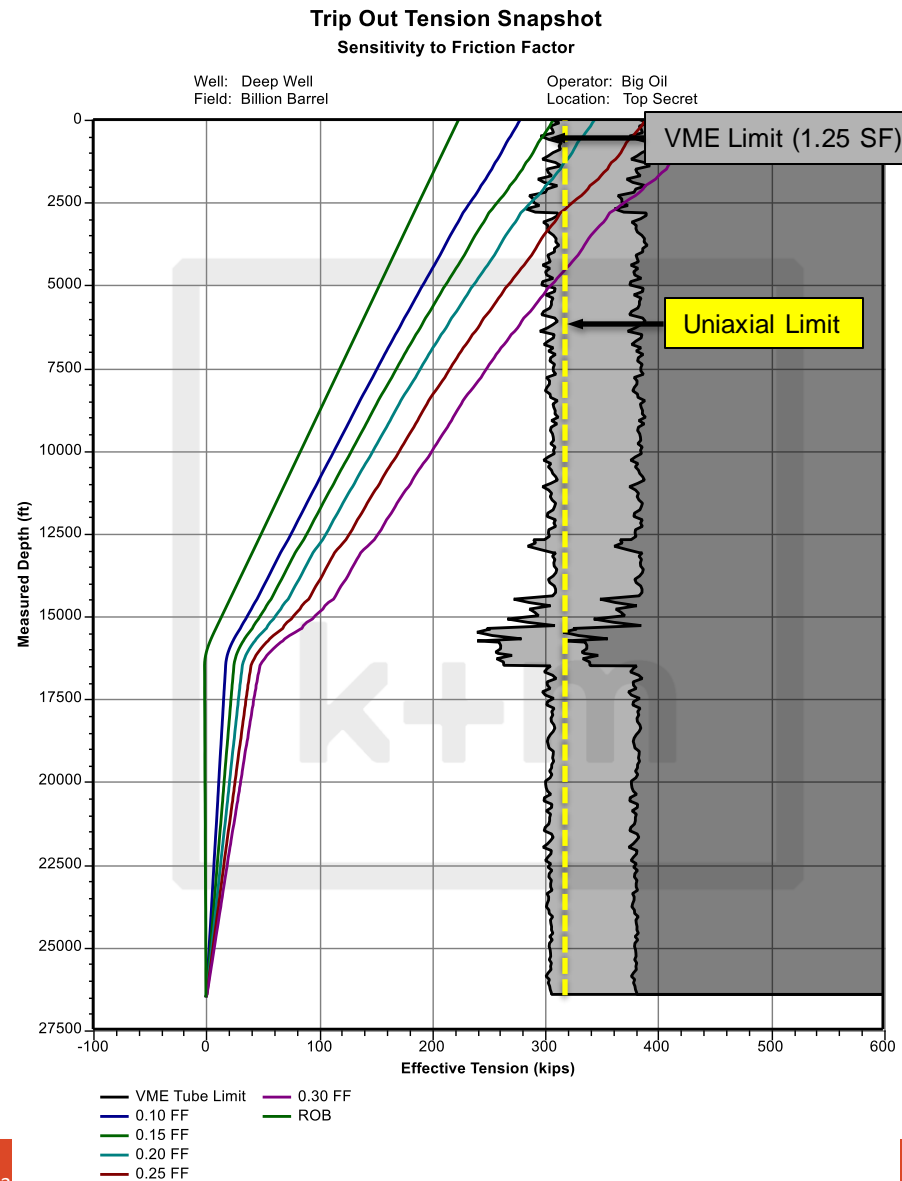
σ_a = Axial stress

σ_b = Bending stress

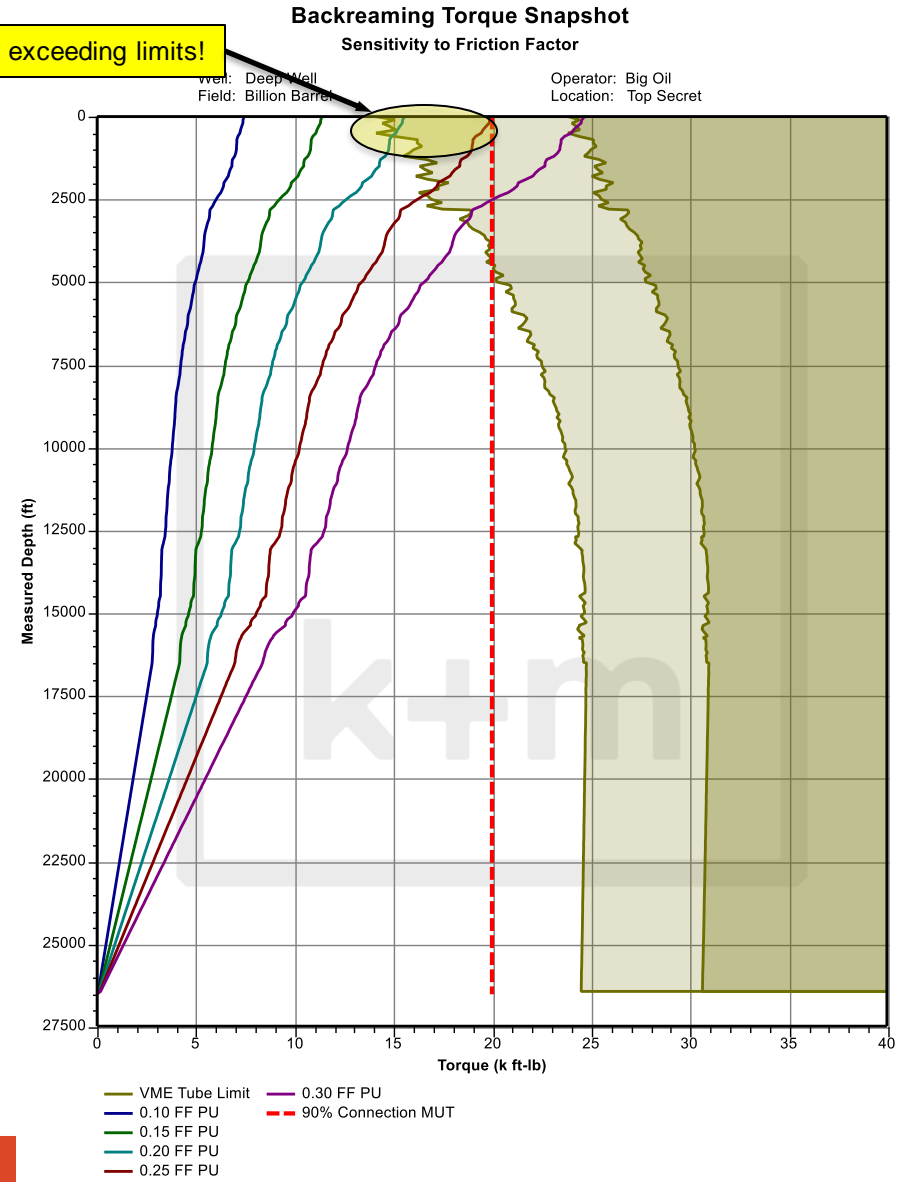
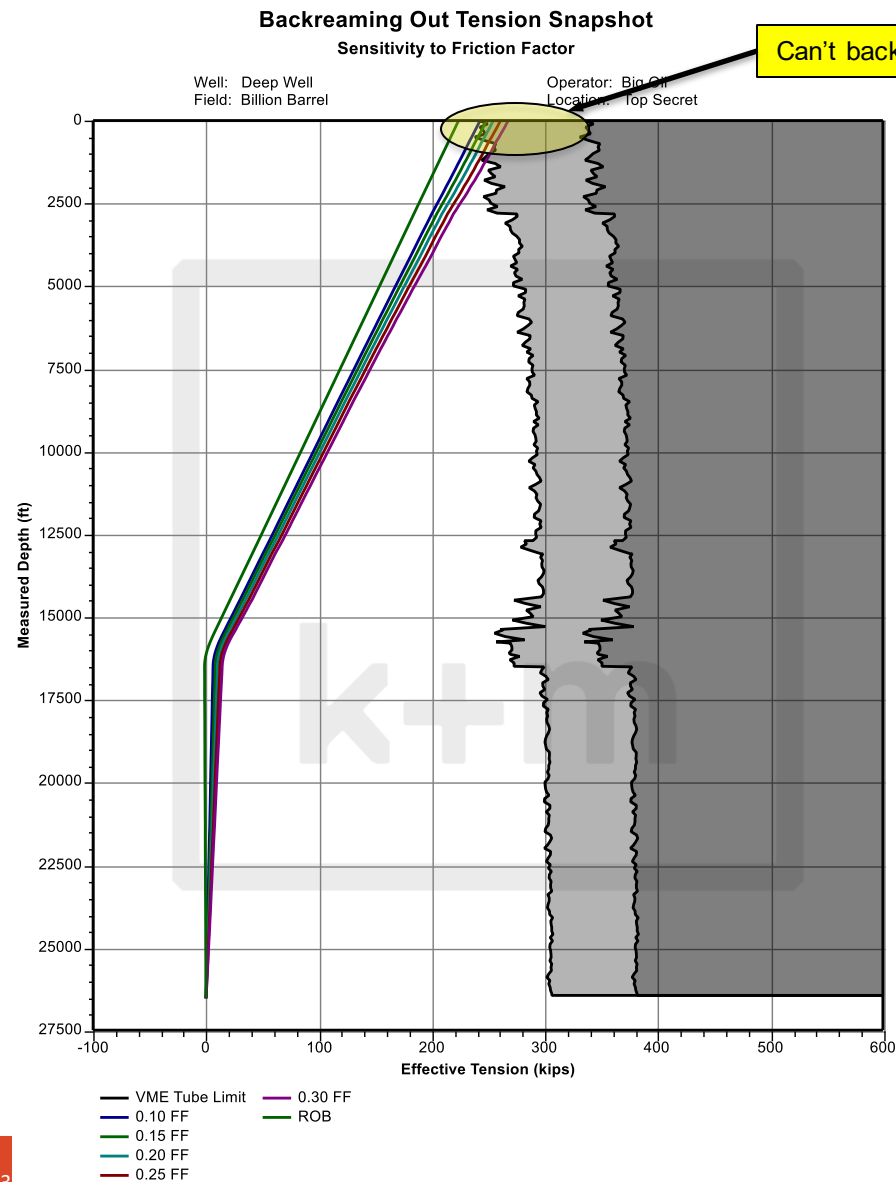
τ = Shear stress



4" 14# Drill Pipe, Tortuosity, VME Limits



4" 14# Drill Pipe, Tortuosity, VME Limits - Backreaming

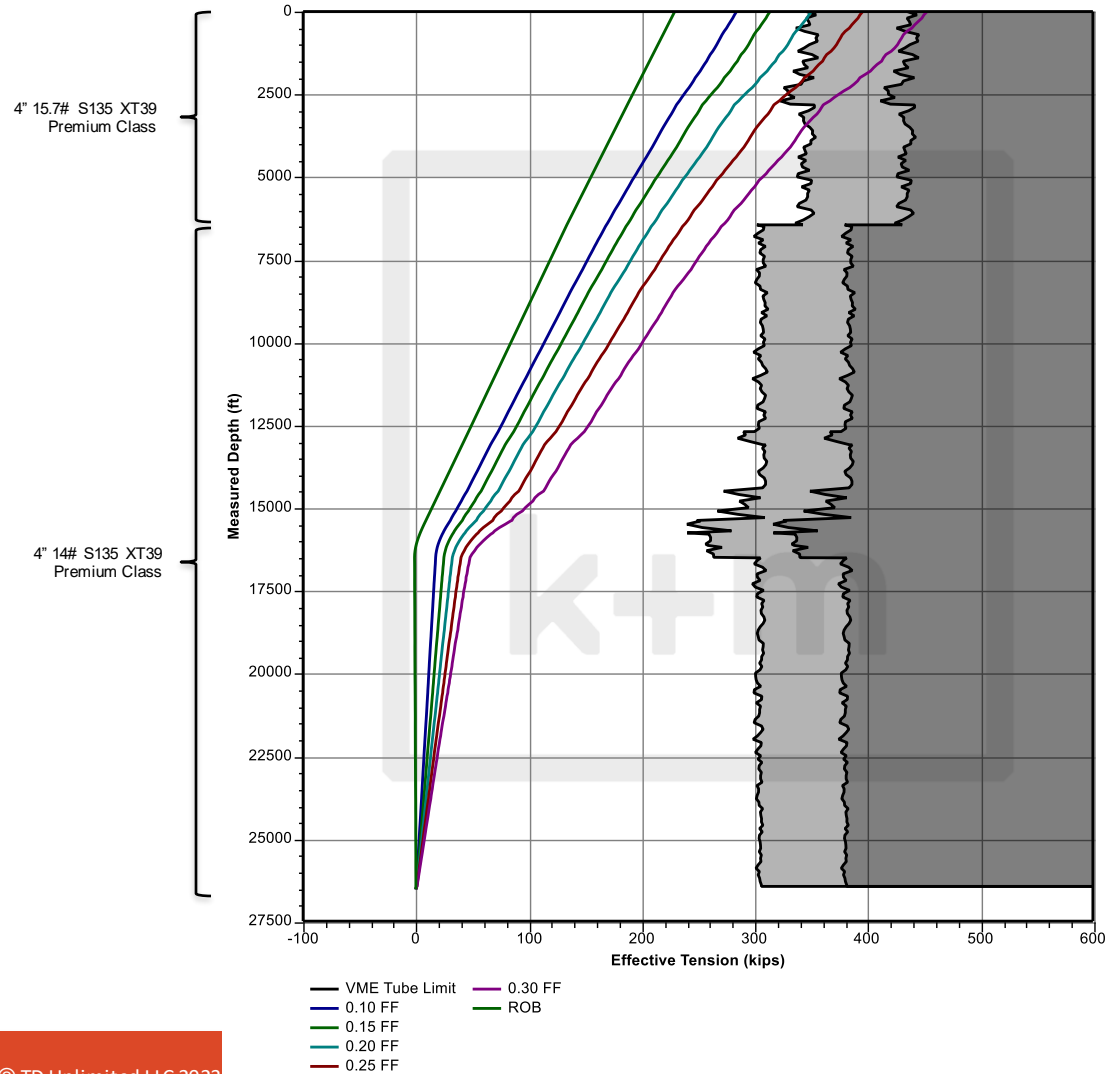


4" 14# x15.7# Drill Pipe, Tortuosity, VME Limits

Trip Out Tension Snapshot

Sensitivity to Friction Factor

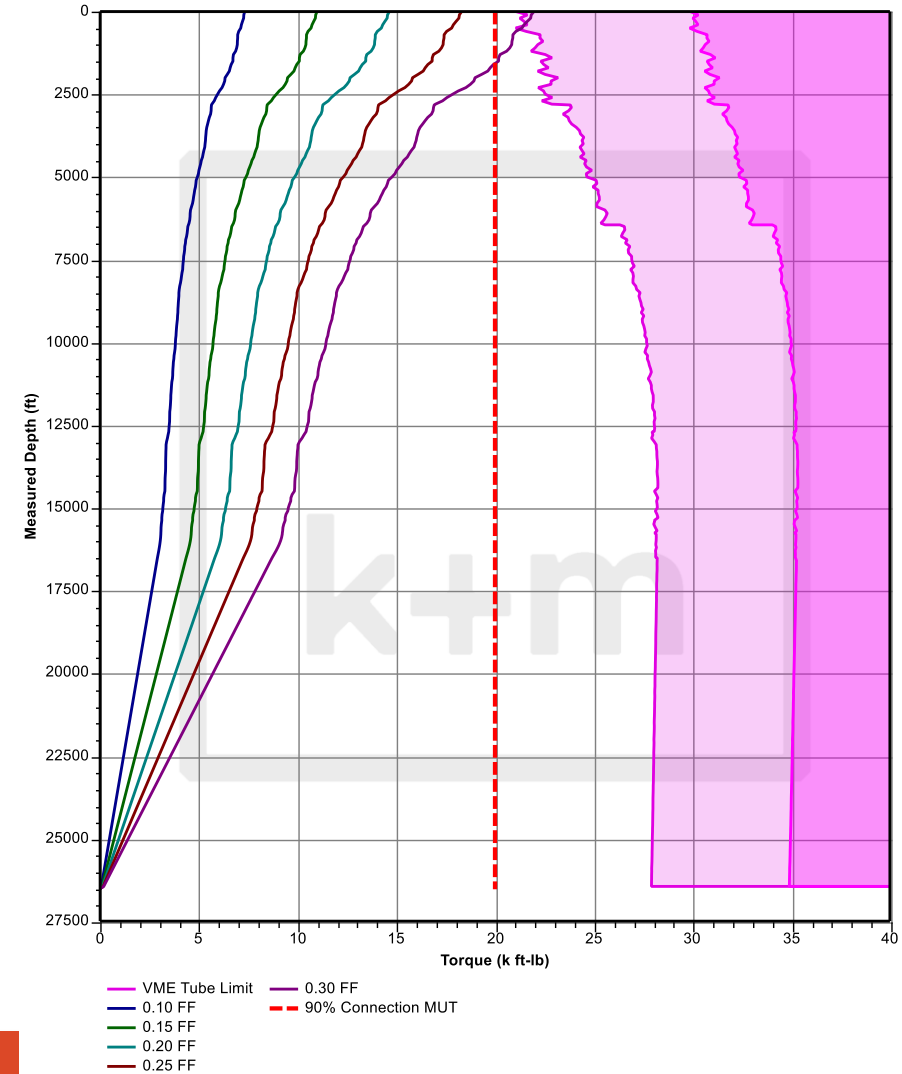
Well: Deep Well
Field: Billion Barrel
Operator: Big Oil
Location: Top Secret



Off Bottom Torque Snapshot

Sensitivity to Friction Factor

Well: Deep Well
Field: Billion Barrel
Operator: Big Oil
Location: Top Secret



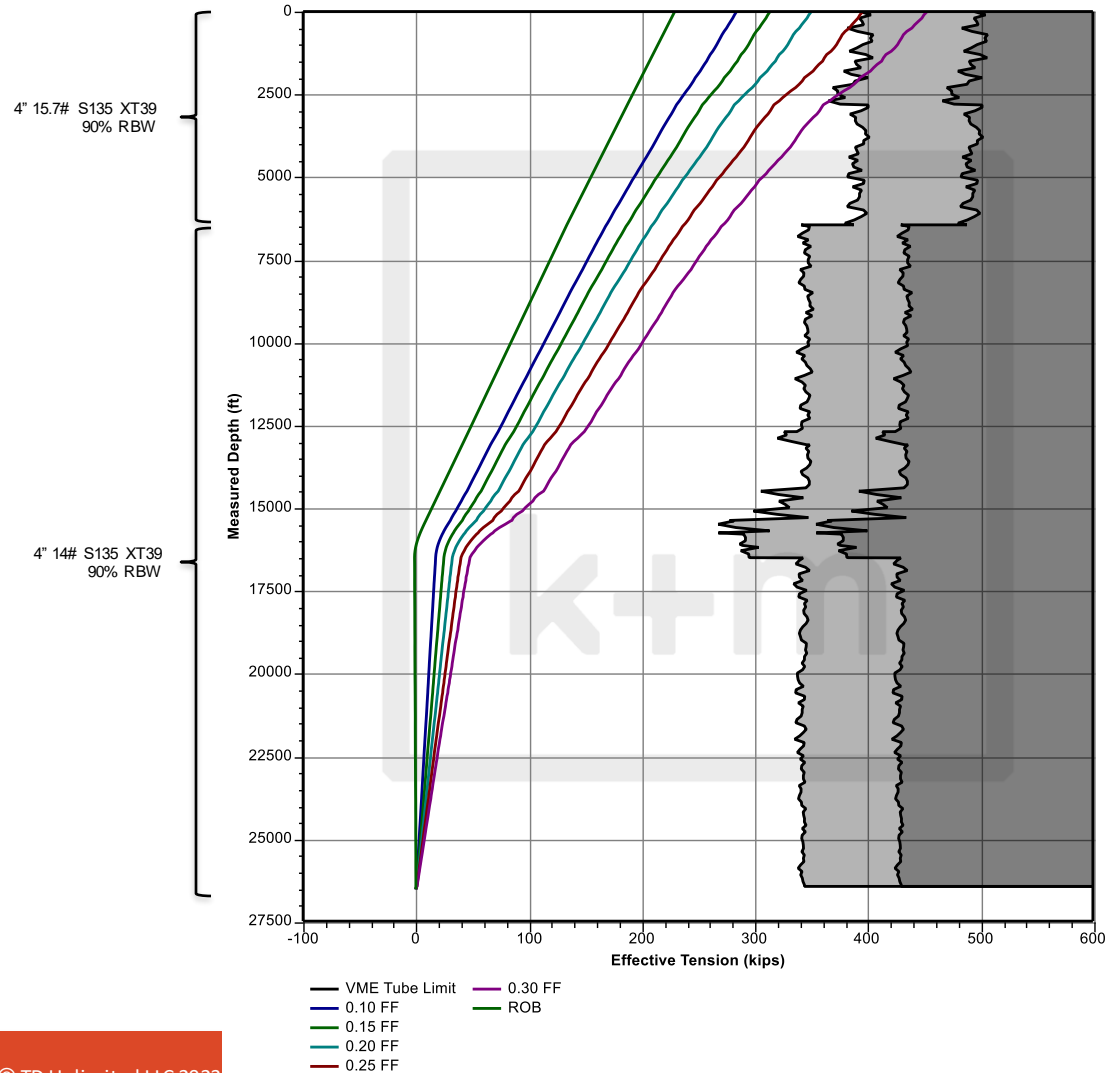
4" 14# x15.7#, 90% RBW Tortuosity, VME Limits

Trip Out Tension Snapshot

Sensitivity to Friction Factor

Well: Deep Well
Field: Billion Barrel

Operator: Big Oil
Location: Top Secret

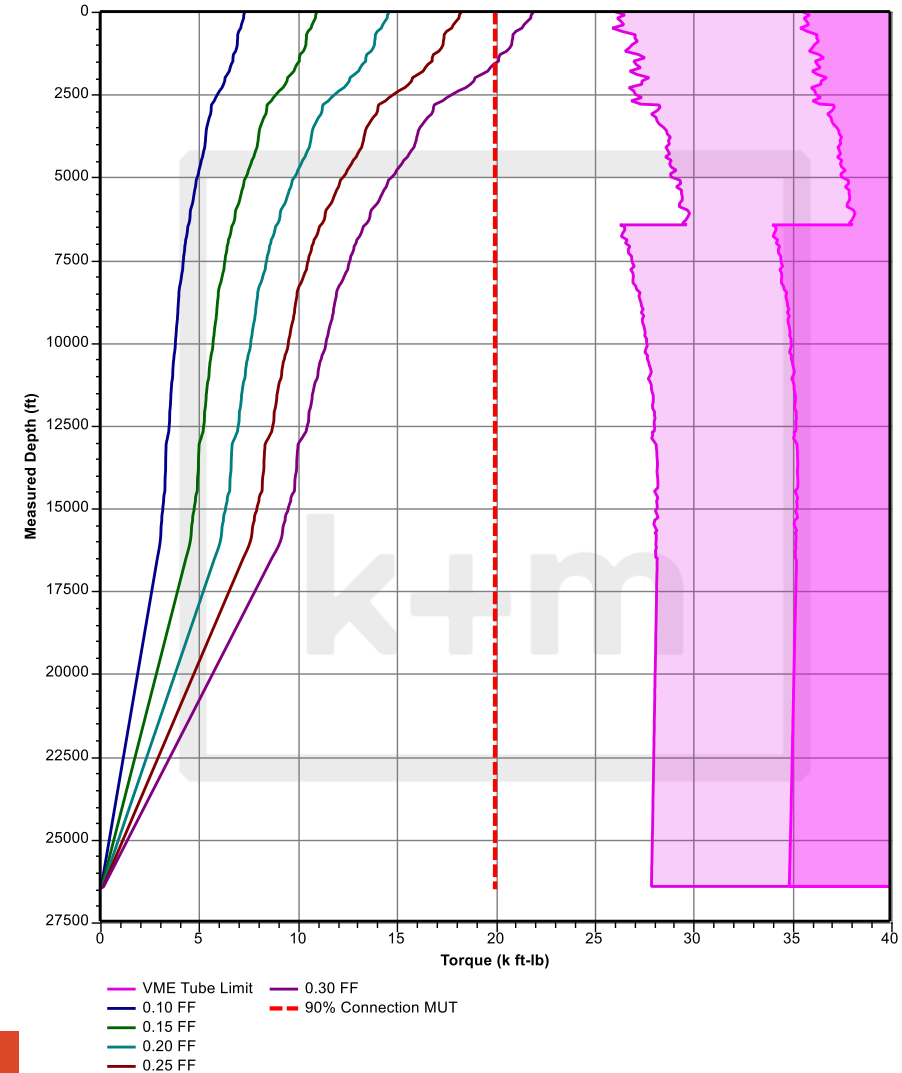


Off Bottom Torque Snapshot

Sensitivity to Friction Factor

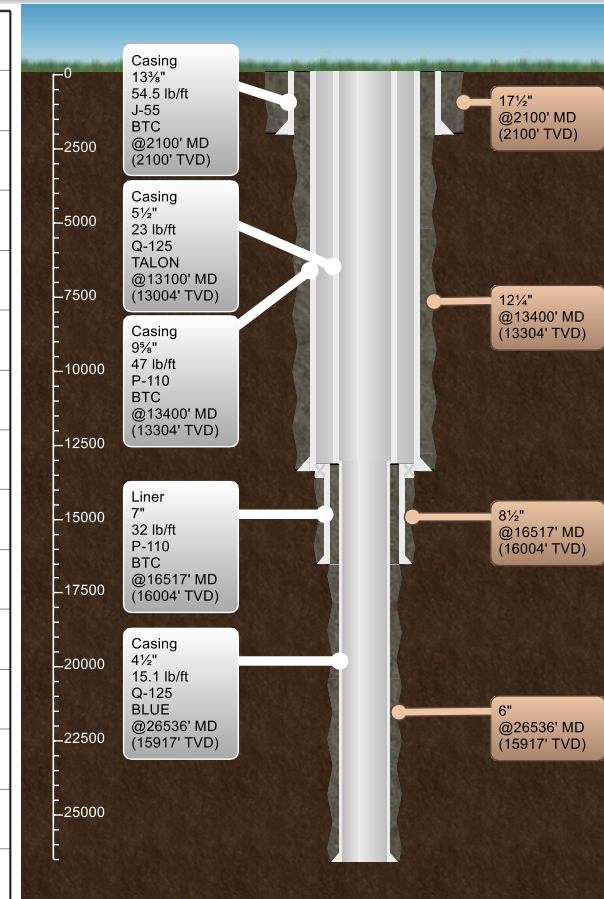
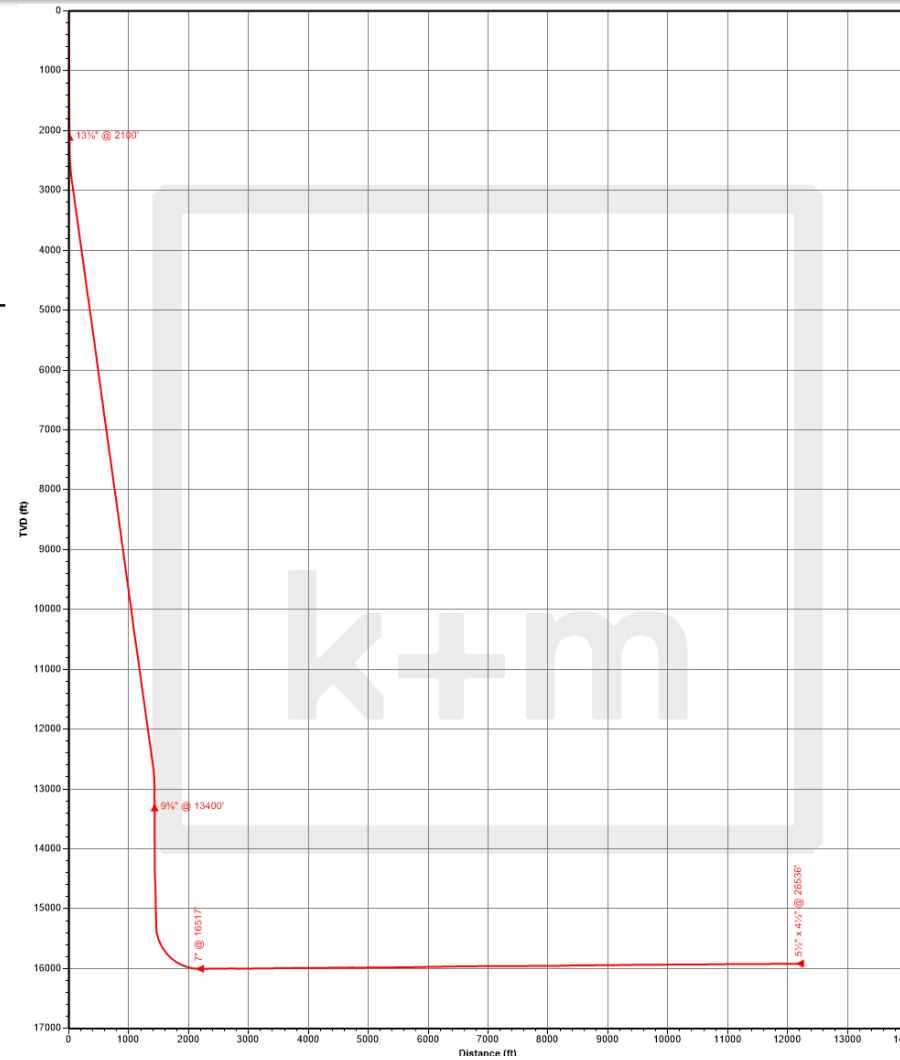
Well: Deep Well
Field: Billion Barrel

Operator: Big Oil
Location: Top Secret



Design Change!

- Why are we setting 9 $\frac{5}{8}$ " so shallow?
 - That's what the vertical offsets wells did 40 years ago
 - Offsets all used WBM and rock bits
 - All the offsets used the same mud weight from 2000' - 13000'
- What if we extend 12 $\frac{1}{4}$ " to 13,000' and run 7" as a liner rather than a long string?
 - Large annulus above the 7" liner top allows us to use a tapered string to drill the lateral
 - Also allows for tapered production casing



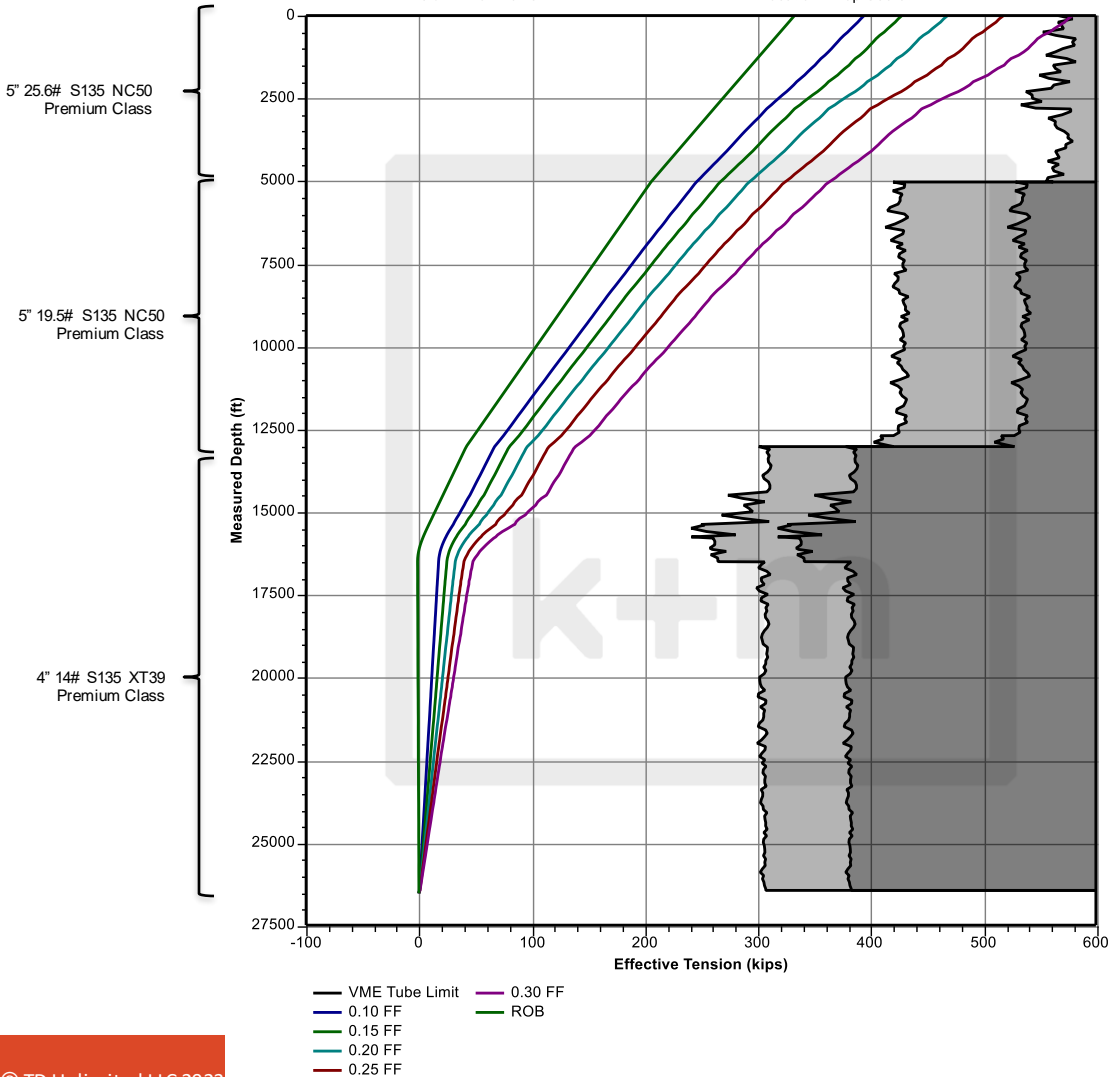
5"x4", Tortuosity, VME Limits

Trip Out Tension Snapshot

Sensitivity to Friction Factor

Well: Deep Well
Field: Billion Barrel

Operator: Big Oil
Location: Top Secret

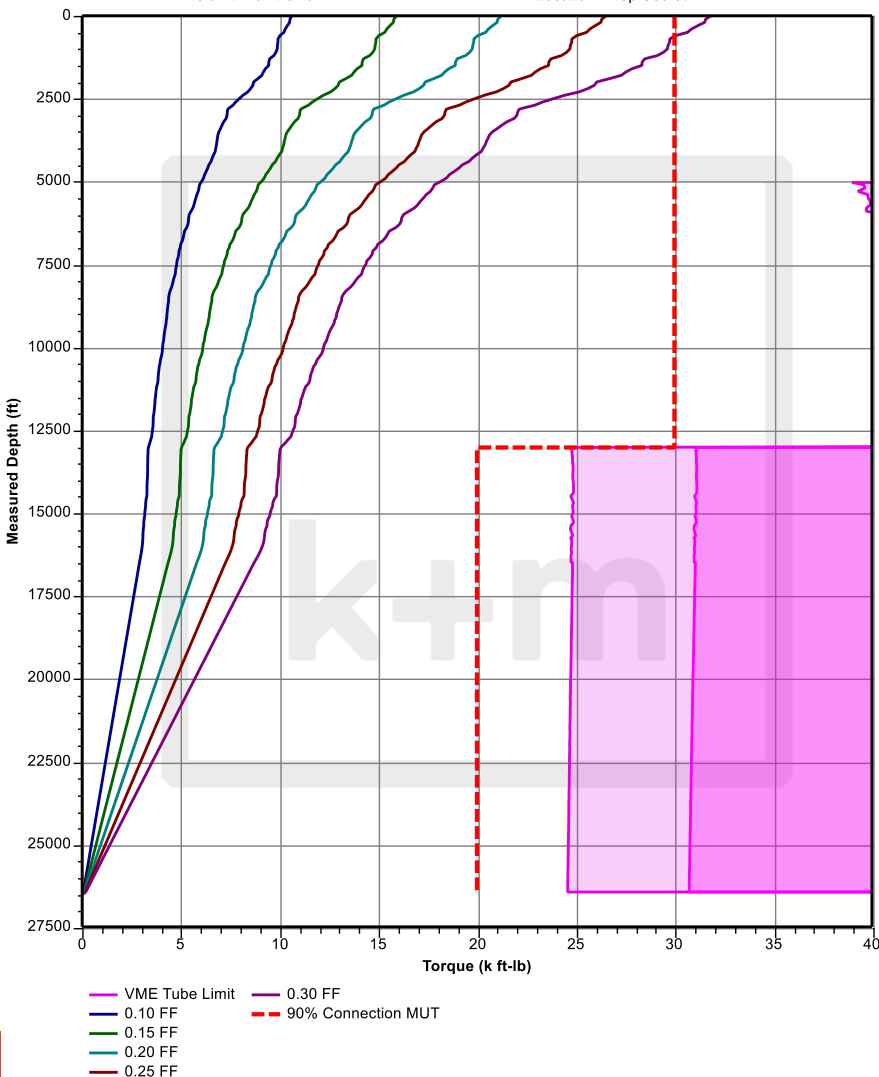


Off Bottom Torque Snapshot

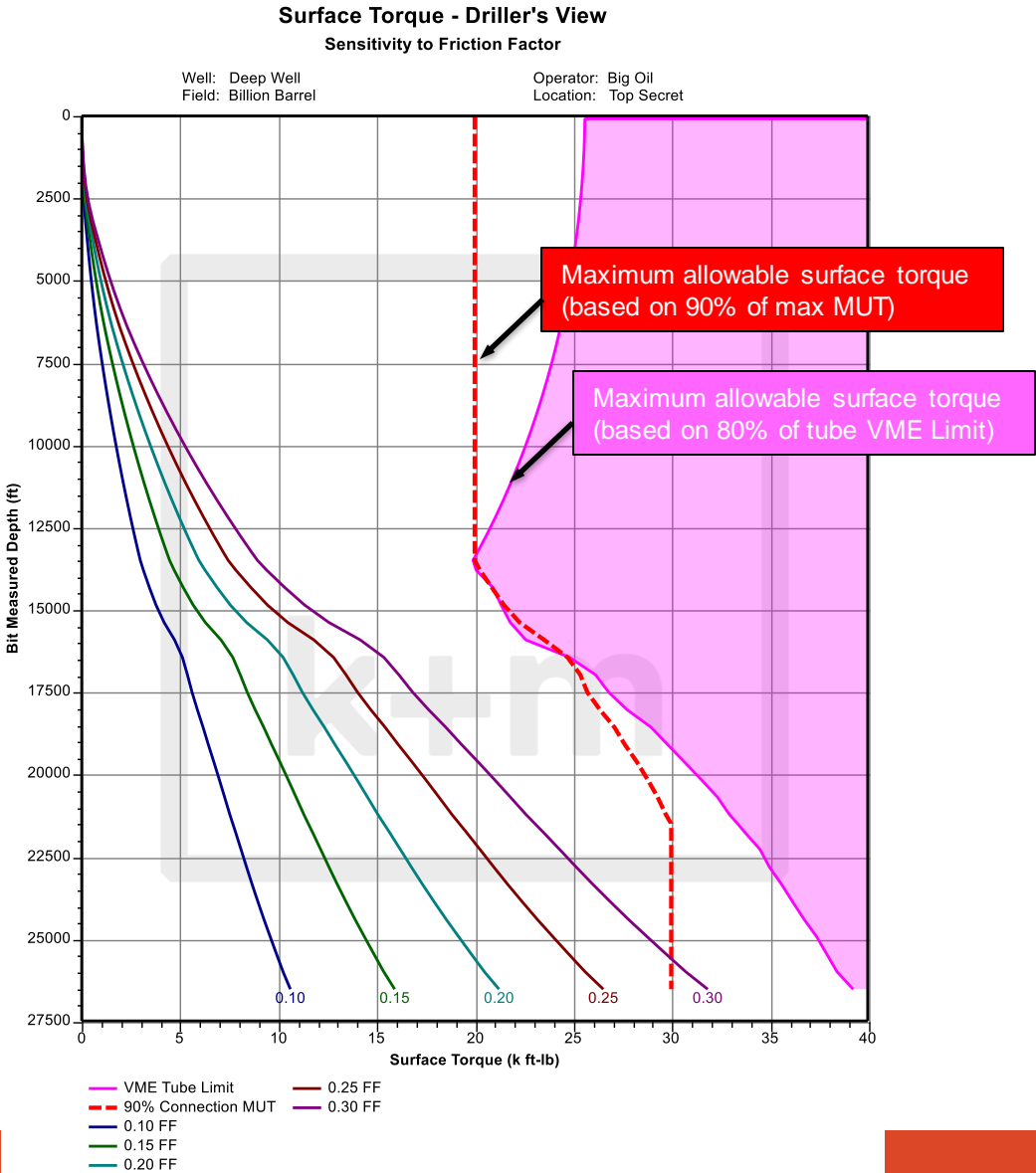
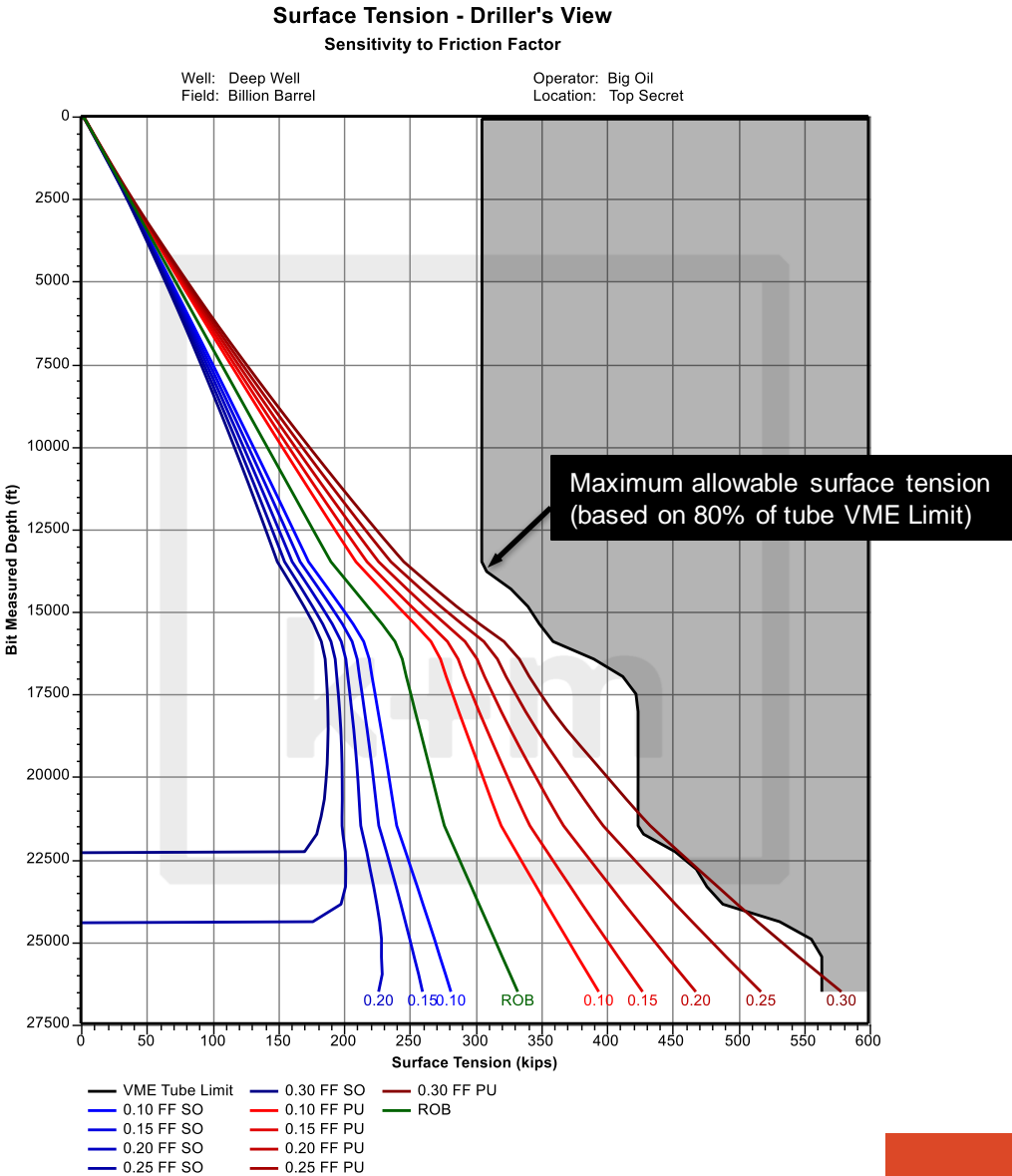
Sensitivity to Friction Factor

Well: Deep Well
Field: Billion Barrel

Operator: Big Oil
Location: Top Secret



Driller's View



Benefits of Design Change

- Adequate torque and tension margins drilling the lateral
- Save \$1MM on casing
- Accommodates 5½"x4½" production casing rather than 4½" liner or 4½" long string
 - Frac down 4½" long string severely limits rate due to friction pressure
 - Running 4½" as a liner and frac down 7" is risky due to wear on 7", and liner top integrity
- Better hydraulics, ECD, and swab/surge drilling and running/cementing casing
- Frac cleanout with 2⅞"x2⅜" work string (wouldn't have been possible with all 2⅜")

Recommended Process

1. Verify pipe condition (New, 90% RBW, Premium)
2. Agree upon minimum design factors
 - 1.20-1.25 is typical for VME or Tension
 - 1.0-1.25 is typical for Tool Joint Makeup Torque
3. Apply artificial tortuosity to planned wellpath (based on offsets)
4. Calculate VME limit for tube tension and torque
5. Base FF assumptions on offsets / analogs
6. Make allowances for torque at bit and overpull

Q&A