

## API 6A Cylindrical Pressure Vessel Calculation

Calculation for pressure vessels subjected to stress from internal pressure, axial loads and an external bending moment.

### Material Data:

Material := "API"

Input "API" for a API Standard Material  
Input "NS" for a Non-Standard API Material

$S_y := 80000 \text{ psi}$

Specified Minimum Yield Strength @ Design Temperature

$S_{yt} := 80000 \text{ psi}$

Specified Minimum Yield Strength @ Test Temperature

$S_{ut} := 95000 \text{ psi}$

Specified Ultimate Tensile Strength

### Design Stress Intensity at Operating Condition

$$S_m := \begin{cases} \min\left(\frac{2}{3} \cdot S_y, \frac{1}{2} \cdot S_{ut}\right) & \text{if Material = "NS"} \\ \frac{2}{3} \cdot S_y & \text{otherwise} \end{cases} = 53333.3 \text{ psi}$$

### Maximum Stress Intensity at Hydrotest Condition

$$S_t := \begin{cases} \min\left(\frac{5}{6} \cdot S_{yt}, \frac{2}{3} \cdot S_{ut}\right) & \text{if Material = "NS"} \\ \frac{5}{6} \cdot S_{yt} & \text{otherwise} \end{cases} = 66666.7 \text{ psi}$$

### Maximum Combined Primary and Secondary Stress Intensity

$$S_s := \begin{cases} \min(2 \cdot S_y, 1 \cdot S_{ut}) & \text{if Material = "NS"} \\ 2 \cdot S_y & \text{otherwise} \end{cases} = 160000 \text{ psi}$$

### Pressures:

$$P_w := 15000 \cdot \text{psi}$$

Working Pressure

$$P_t := 1.5 \cdot P_w = 22500 \text{ psi}$$

Hydrotest Pressure

### Geometrical Data:

$$\text{OD} := 10.095 \cdot \text{in} = 10.095 \text{ in}$$

Minimum Outer Diameter of Pressure Vessel

$$\text{ID} := 7.395 \cdot \text{in}$$

Maximum Inside Diameter of Pressure Vessel

### Calculation of Variables

$$R_o := \frac{\text{OD}}{2} = 5.048 \text{ in}$$

Max Outside Radius

$$t := R_o - \frac{\text{ID}}{2}$$

$$t = 1.35 \text{ in}$$

Wall Thickness

$$R_i := \frac{\text{ID}}{2}$$

$$R_i = 3.697 \text{ in}$$

Inside Radius

$$A_x := \pi (R_o^2 - R_i^2)$$

$$A_x = 37.089 \text{ in}^2$$

Cross-Sectional Area

$$R_m := R_o - \frac{t}{2} = 4.372 \text{ in}$$

Radius at Midsurface

$$\sqrt{R_m \cdot t} = 2.43 \text{ in}$$

Max Distance for Local Membrane Stress

### External Loading

$$F_a := 0 \text{ lbf}$$

Axial Force (negative denotes compression)

$$M := 0 \cdot \text{ft} \cdot \text{lbf}$$

Bending Moment

### Membrane Stresses at Operating Pressure

$$P_{mh} := \frac{P_w \cdot R_i}{t} = 41083.3 \text{ psi}$$

Membrane Hoop Stress

$$P_{mr} := \frac{-P_w}{2} = -7500 \text{ psi}$$

Membrane Radial Stress

$$P_{mz1} := \frac{(P_w \cdot \pi \cdot R_i^2 + F_a)}{A_x} + 4 \cdot M \cdot \frac{\left(R_i + \frac{t}{2}\right)}{\pi(R_o^4 - R_i^4)}$$

$$P_{mz1} = 17370.6 \text{ psi}$$

Membrane Axial Stress on Tension Side

$$P_{mz2} := \frac{(P_w \cdot \pi \cdot R_i^2 + F_a)}{A_x} - 4 \cdot M \cdot \frac{\left(R_i + \frac{t}{2}\right)}{\pi(R_o^4 - R_i^4)}$$

$$P_{mz2} = 17370.6 \text{ psi}$$

Membrane Axial Stress on Compression Side

### Max Stress Intensity at Operating Pressure

$$D_1 := |P_{mh} - P_{mr}| = 48583.3 \text{ psi}$$

$$D_2 := |P_{mh} - P_{mz1}| = 23712.8 \text{ psi}$$

$$D_3 := |P_{mz1} - P_{mr}| = 24870.6 \text{ psi}$$

$$D_4 := |P_{mh} - P_{mz2}| = 23712.8 \text{ psi}$$

$$D_5 := |P_{mz2} - P_{mr}| = 24870.6 \text{ psi}$$

$$P_m := \max(D_1, D_2, D_3, D_4, D_5)$$

Allowable

$$S_m = 53333.3 \text{ psi}$$

$$P_m = 48583.3 \text{ psi}$$

Primary Membrane Stress Intensity at Operating Condition

### Max Primary Plus Secondary Stress Intensity at Operating Pressure

$$P_s := \frac{2 \cdot P_w \cdot \left(\frac{R_o}{R_i}\right)^2}{\left(\frac{R_o}{R_i}\right)^2 - 1}$$

Equation 4-222 (2) from ref 2

Allowable

$$S_s = 160000 \text{ psi}$$

$$P_s = 64741.1 \text{ psi}$$

Primary Plus Secondary Stress Intensity at Operating Condition

### Max Stress Intensity at Hydrotest Pressure

$$P_{mt} := \frac{P_t \cdot R_i}{t} + \frac{P_t}{2}$$

Equation 4-222 (1) from ref 2

### Allowable

$$S_t = 66666.7 \text{ psi}$$

$$P_{mt} = 72875 \text{ psi}$$

Primary Membrane Stress Intensity at Hydrotest Condition

### Conclusions:

$$\text{Stress}_{\text{Operating}} := \text{if}(P_m \leq S_m \wedge P_s \leq S_s, \text{"Pass"}, \text{"Fail"})$$

$$\text{Stress}_{\text{Operating}} = \text{"Pass"}$$

$$\text{Stress}_{\text{Hydrotest}} := \text{if}(P_{mt} \leq S_t, \text{"Pass"}, \text{"Fail"})$$

$$\text{Stress}_{\text{Hydrotest}} = \text{"Fail"}$$

### References:

1. Design Handbook for API 6A / 16A / 17D Equipment, John H Fowler, Ninth Edition 2011.
2. 2004 ASME Boiler and Pressure Vessel Code, Section VIII Division 2 - Alternative Rules, ASME, New York, 2004
3. American Petroleum Institute, Specification for Wellhead and Christmas Tree Equipment, API Specification 6A, Nineteenth Edition / ISO 10423:2003 (Modified), API, Washington DC, 2004