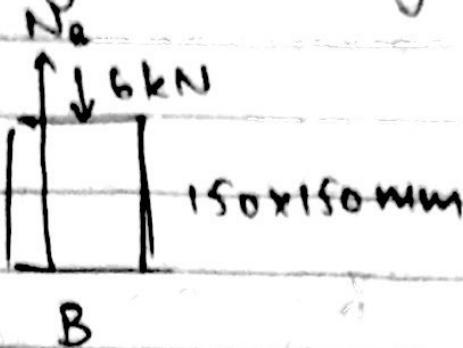


UES010 (Solids & Structures)

Test 1

Bearing stress by force at A, B & C

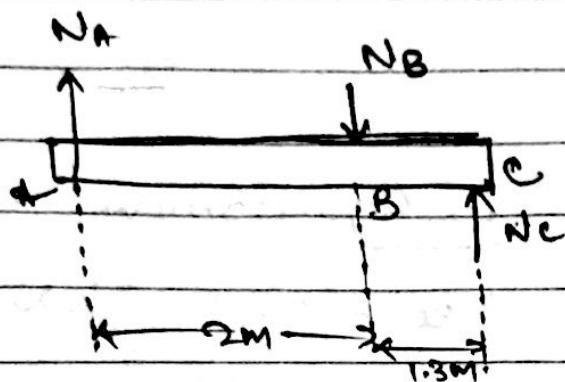


$$N_B = 6 \text{ kN}$$

$$\sigma = \frac{P}{A}$$

$$\sigma = \frac{6 \times 10^3}{150 \times 150 \times 10^{-6}}$$

$$\sigma_B = 0.266 \times 10^6 \text{ N/m}^2 \\ = 0.266 \text{ MPa.}$$



$$N_A + N_C = N_B$$

$$N_A + N_C = 6 \text{ kN.}$$

Moment abt A:

$$2N_B = 3.3N_C$$

$$\frac{2(6 \text{ kN})}{3.3} = N_C$$

$$N_C = 3.636 \text{ kN.}$$

Also

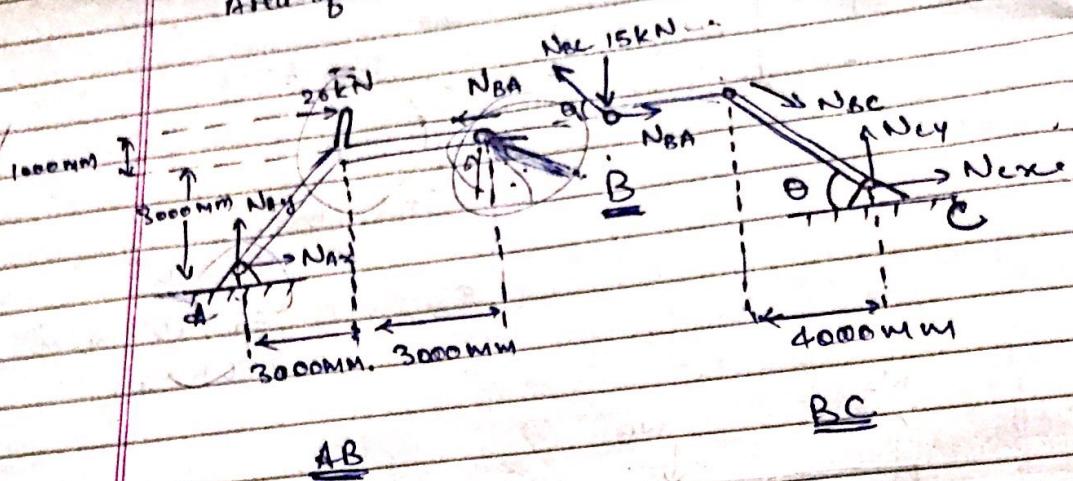
$$N_A + N_C = 6 \text{ kN}$$

$$\therefore N_A = 6 - N_C$$

$$= 6 - 3.636$$

$$= 2.36 \text{ kN}$$

- ② Intermediate hinge at B
 reaction at A & C and at B
 F.B.D. of AB & BC, Normal stress in BC
 Area of cross-section of BC = 400 mm^2



$$\tan \theta = \frac{3}{4}$$

$$\Rightarrow \theta = 36.86^\circ$$

$$\sin \theta = 0.6$$

$$\cos \theta = 0.8$$

Force Equilibrium at B

$$N_{BC} \sin \theta = 15 \text{ kN}$$

$$N_{BC} = \frac{15 \times 10^3}{0.6}$$

$$\underline{N_{BC} = 25 \text{ kN}}$$

$$\text{Also } N_{BC} \cos \theta = N_{BA}$$

$$25 \times 0.8 = N_{BA}$$

$$\underline{N_{BA} = 20 \text{ kN}}$$

Torque Balance?

Force equilib. At AB

$$N_{AX} + 20 = N_{BA}$$

$$N_{AX} = 0$$

$$N_{AY} = 0$$

Force eq. at BC

$$N_{BC} \cos \theta + N_{CX} = 0$$

$$N_{CX} = -25 \times 0.8$$

$$\cancel{20} - 20 \text{ kN}$$

$$N_{BC} \sin \theta = N_{CY}$$

$$N_{CY} = 25 \times 0.6$$

$$= 15 \text{ kN}$$

DELL

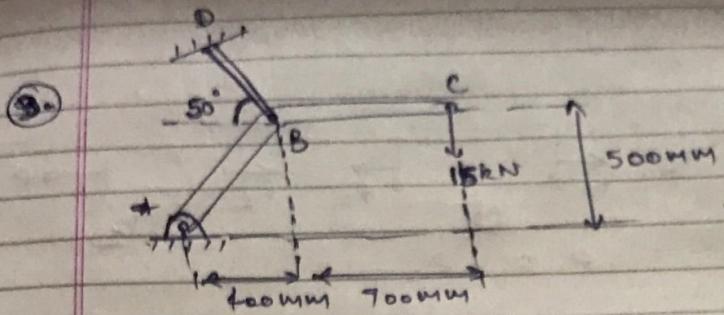
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Key
→ N
C

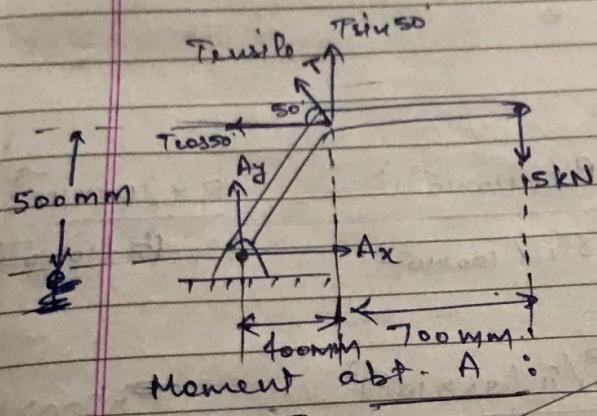
B

$$\text{Normal Stress ABC} = \frac{25 \times 10^3}{400 \times 10^{-6}} = 62.5 \text{ MPa}$$



Tensile stress
on BD?
Shearing stress
in fl. at A.

Diam. (flange) = 20 mm
(pin) = 10 mm
FBD of ABC,



Force equilibrium:

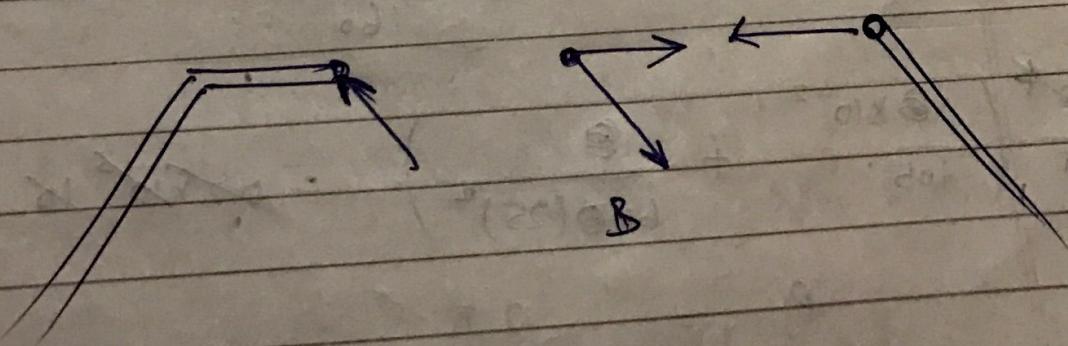
$$A_y + 0.766 T = 15 \text{ kN}$$

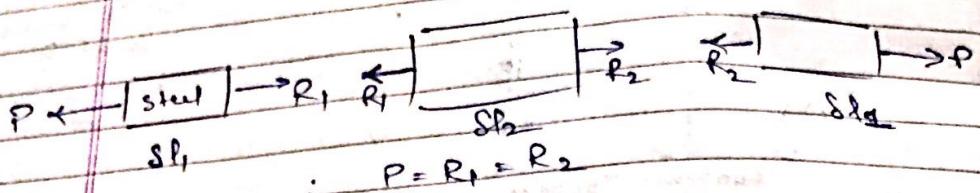
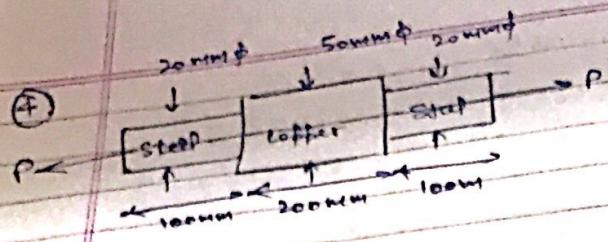
$$A_x = T (0.64)$$

~~$$500 \times 10^{-3} (0.64 T) + (0.766 T)(400 \times 10^{-3}) = 15 \times 10^3 (1100 \times 10^{-6})$$~~

$$3.2 T + 3.064 T = 165 \text{ kN}$$

$$\underline{T = 26.34 \text{ kN}}$$





$$E_{\text{Steel}} = 210 \text{ GPa}$$

$$E_{\text{Timber}} = 120 \text{ GPa}$$

$$210 \times 10^9 = \frac{P / \pi (10 \times 10^{-3})^2}{8l_1 / 100 \times 10^{-3}} = \frac{P \times 100 \times 10^{-3}}{8l_1 (\pi (10 \times 10^{-3})^2)}$$

$$120 \times 10^9 = \frac{P / \pi (25 \times 10^{-3})^2}{8l_2 / 200 \times 10^{-3}} = \frac{P \times 200 \times 10^{-3}}{8l_2 (\pi (25 \times 10^{-3})^2)}$$

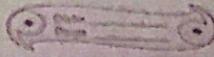
$$28l_1 + 8l_2 = 0.2 \text{ mm}$$

$$2 \left(\frac{P \times 10^4 \times 10^{-6}}{210 \times 10^9 (\pi) (10)^4} \right) + \frac{P \times 2 \times 10^{-9+6}}{120 \times 10^9 (\pi) (25 \times 10^{-3})^2} = 0.2 \times 10^{-3}$$

$$\frac{P}{\pi} \left(\frac{B \times 10^{-2}}{105} + \frac{B}{60 \times (25)^2} \right) = 0.2 \times 10^{-3} \times 10^{-2}$$

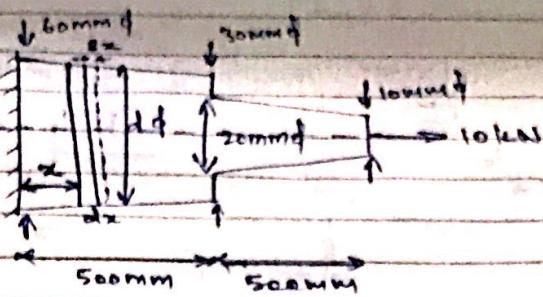
$$P = \frac{2 \pi}{\frac{10^{-2}}{105} + \frac{1}{625 \times 60}} = 51.541 \text{ kN}$$

0.12m



$$P = 10 \text{ kN}$$

$$E = 205 \text{ GPa}$$



$$\frac{60-30}{500} = \frac{60-d}{x}$$

$$\left(\frac{30}{500}\right)x = 60-d$$

$$d = 60 - \frac{3x}{50}$$

$$205 \times 10^9 = P / \pi (d/2 \times 10^3)^2$$

$$\frac{\delta x}{dx} = P$$

$$\pi \left(30 - \frac{3x}{100}\right) \times 10^{-6} \times 205 \times 10^9$$

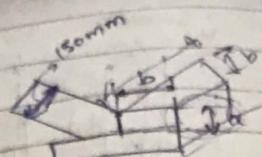
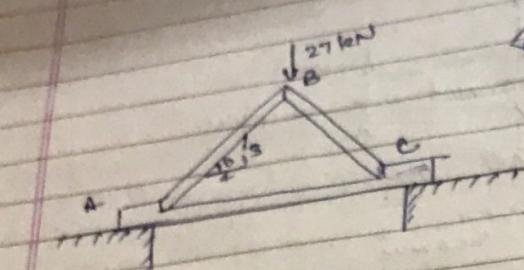
~~$$S = \frac{4PL}{\pi E d_1 d_2}$$~~

~~$$S_{\text{net}} = \delta_1 + \delta_2$$~~

~~$$= \frac{4 \times 10 \times 10^3 \times 500 \times 10^{-3}}{\pi \times 205 \times 10^9 \times 10^3} \left(\frac{1}{60 \times 30} + \frac{1}{30 \times 10} \right)$$~~

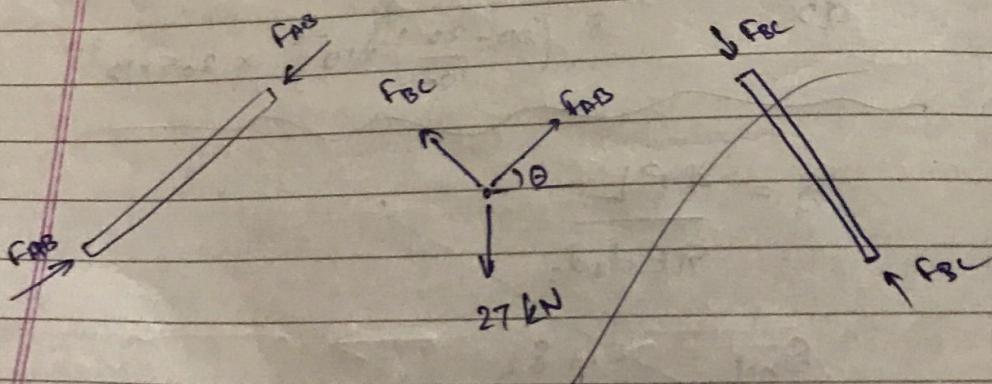
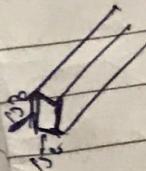
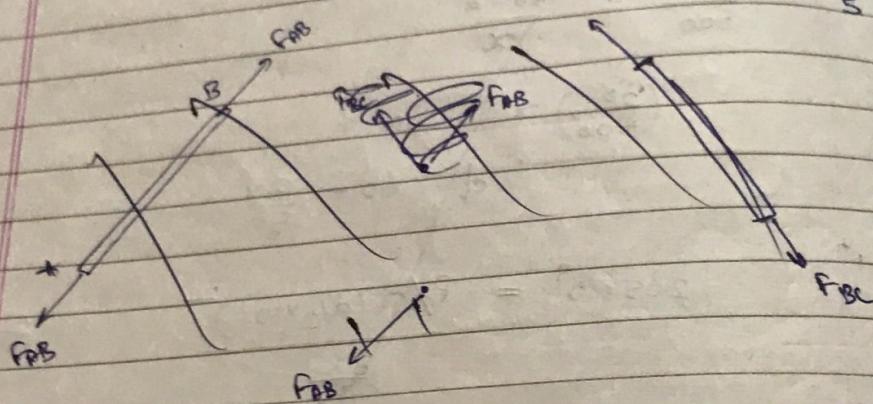
~~$$0.12 \text{ m} = \frac{2}{3} \times \frac{1}{205 \pi} \left(\frac{7}{60} \right)^2 = \frac{4 \times 10 \times 500 \times 10^3}{\pi \times 205} \times \frac{1}{36} \left(\frac{1}{60} + \frac{1}{10} \right)$$~~

⑥ Allowable direct bearing = 6 N/mm^2
 Shear = 1.2 N/mm^2



$$\cos \theta = \frac{4}{5}$$

$$\sin \theta = \frac{3}{5}$$



$$27 = F_{AB} \sin \theta + F_{BC} \sin \theta$$

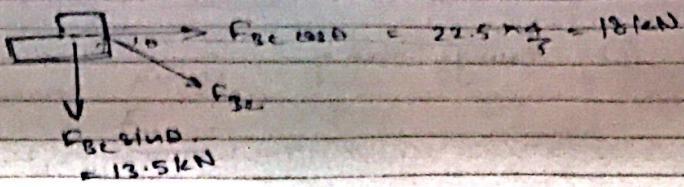
$$F_{AB} \cos \theta = F_{BC} \cos \theta$$

$$2F_{BC} \cancel{\times \frac{3}{5}} = 27$$

$$F_{AB} = F_{BC}$$

$$F_{BC} = 22.5 \text{ kN.}$$

DELL



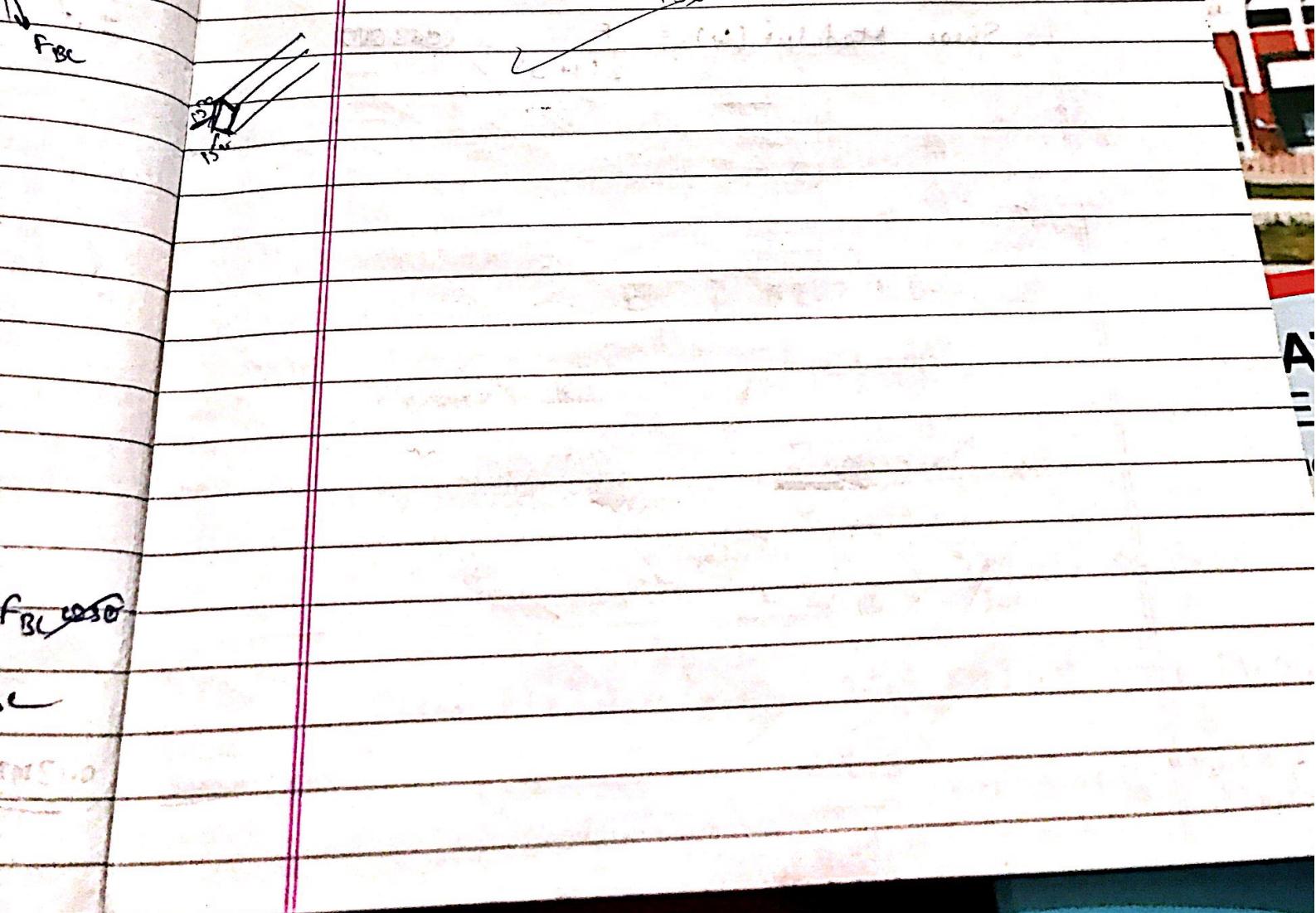
$$b = \frac{18 \times 1000}{150 \times h}$$

$$b = 20 \text{ mm}$$

$$b = \frac{13.5 \times 1000}{150 \times h}$$

$$1.2 = \frac{13.5 \times 1000}{150 \times h}$$

$$h = \frac{135000}{156 \times 12}$$



Tut Sheet 2

① Young's Mod = $\frac{\text{Stress}}{\text{Strain}}$

$$= \frac{25 \times 10^3}{\pi (8 \times 10^{-3})^2}$$

$$\frac{300 \times 10^{-6}}{500 \times 10^{-3}}$$

$$= \frac{25 \times 10^3}{\pi \times 64 \times 10^{-6}} \times \frac{500 \times 10^{-3}}{300 \times 10^{-6}}$$

$$= 207.23 \times 10^{12-3} \text{ N/m}^2$$

$$= 207.23 \times 10^9 \text{ N/m}^2$$

Poisson's Ratio = $\frac{2.4 \times 10^{-6}}{\frac{16}{500}}$
 $= 0.25 - v$

Shear Modulus (G) = $\frac{E}{2(1+v)}$ ~~0.25 - v~~

① Specimen length
 $\phi = 16 \text{ mm}$
 $\delta l = 300 \mu\text{m}$
 $\delta \theta = 24 \mu\text{m}$
Axial load =

~~16mm~~
~~300μm~~
~~24μm~~

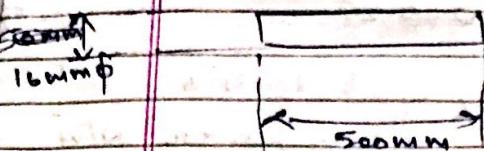
Mod. of

500 mm long
 $\phi = 16 \text{ mm}$
 $\delta L = 300 \mu\text{m}$
 $\delta P = 24 \mu\text{m}$
Axial load = 25 kN

Tilt Street - 2

Tensile stress
Plastic strain
> Gauge load

Mod. of Elastic?
Poisson's Ratio?
Shear Mod?



Poisson's Ratio =
- Contraction
- Extension
- Lateral
- Elongation

Mod. of Elastic = $\frac{\text{Stress}}{\text{Strain}}$

$$\frac{25 \times 1000}{\pi \times (8 \times 10^{-6})^3}$$

2.4

16

$$= \frac{25 \times 1000}{\pi \times 8.64 \times 10^{-16}} \times \frac{6 \times 16}{2.4}$$

$$= 828.93 \times 10^6 \text{ N/m}^2$$

$$E = 0.828 \times 10^9 \text{ Pa}$$

$$\text{Poisson's Ratio} = + \frac{2.4 \times 10^{-6}}{16}$$

$$\frac{300 \times 10^{-6}}{500}$$

$$v = \frac{2.4}{16} \times \frac{500}{300} = 0.25$$

$$\text{Shear Modulus, } G \Rightarrow 2G(1+v) = E = 3K(1-2v)$$

$$G = \frac{E}{2(1+v)} = \frac{0.828}{2(1.25)} = 0.3312 \text{ GPa}$$

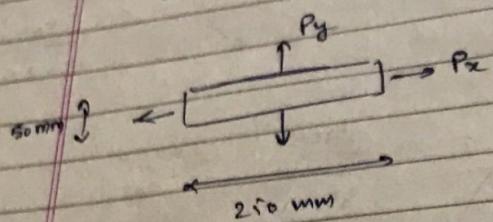
Bulk
Mod.

- (2) $50 \times 250 \times 10 \text{ mm}$
 Stresses along edges?
 a) $P_x = 100 \text{ kN}$
 b) $P_y = 200 \text{ kN}$

Change in thickness?

- b) $P_x = ?$ for some change.

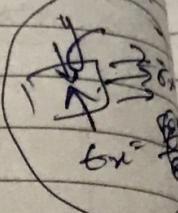
$$\bullet E = 200 \text{ GPa} \quad v = 0.25$$



$$E = 200 \text{ GPa}$$

$$= 200 \times 10^9 \text{ N/mm}^2$$

$$= 200 \text{ kN/mm}^2$$



$$\epsilon_x = \frac{\sigma_x}{E} - v \left(\frac{\sigma_x}{E} + \frac{\sigma_y}{E} \right)$$

$$= 0 - \frac{0.25}{200 \times 10^9 \times 10} \left(\frac{100 \times 10^3}{10 \times 50} + \frac{200 \times 10^3}{250 \times 10} \right)$$

$$= -\frac{0.25}{20 \times 50} \left(1 + \frac{2}{5} \right) = -3.5 \times 10^{-4}$$

$$(\Delta l)_x = -3.5 \times 10^{-4} \times 10 = -3.5 \times 10^{-3} \text{ mm}$$

$$\epsilon_x = -v \frac{\sigma_x}{E}$$

$$\sigma_x = \frac{-3.5 \times 10^{-4} \times 200 \times 10^3}{0.25 \times 10} = \frac{P_x}{50 \times 10}$$

$$P_x = \frac{3.5 \times 200 \times 50}{0.25} = 140 \text{ kN}$$

$$E = 210 \text{ GPa}$$

$$v = 0.3$$

$$\sigma_x = 150 \text{ MPa}$$

$$\bullet \epsilon_y = \frac{\sigma_y}{E}$$

$$\epsilon_y = 0$$

$$\epsilon_x =$$

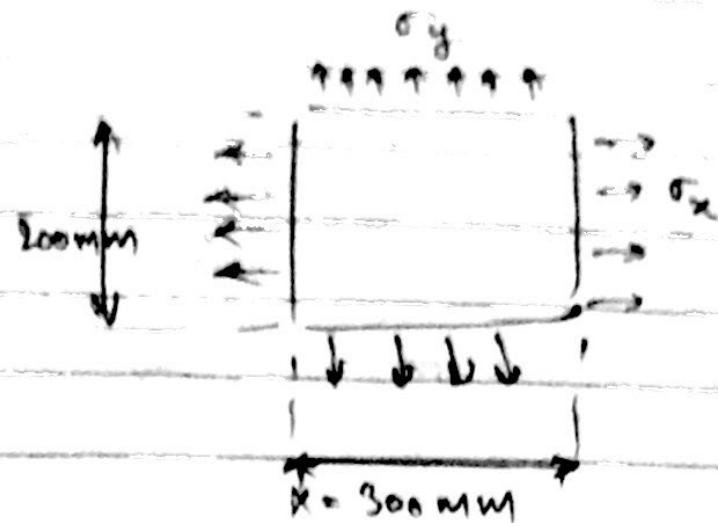
③

$$E = 210 \text{ GPa}$$

$$\nu = 0.3$$

$$\sigma_x = 150 \text{ MPa}$$

300x200x10 mm



$$\epsilon_y = \frac{\sigma_y}{E} - \nu \left(\frac{\sigma_x}{E} \right)$$

$$\epsilon_y = 0 \Rightarrow \sigma_y = \nu \cdot \sigma_x \\ = 0.3 (150) \\ = 45 \text{ MPa}$$

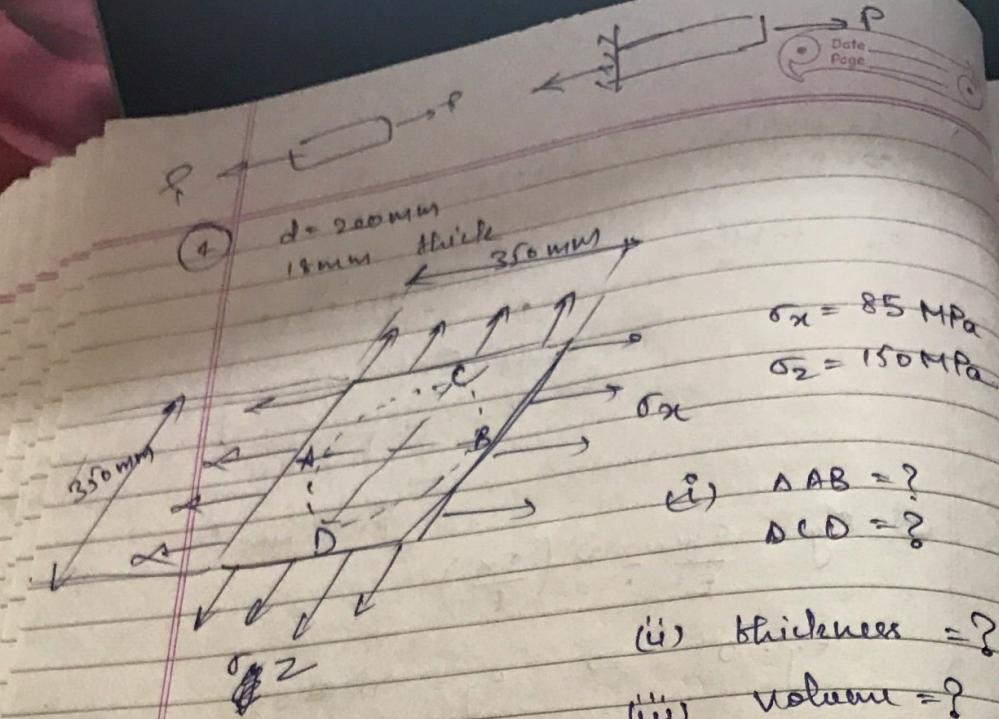
$$\epsilon_x = \frac{\sigma_x}{E} - \nu \left(\frac{\sigma_y}{E} \right)$$

$$= \frac{1}{E} (150 - 0.3 (45)) = 0.65 \times 10^{-3}$$

$$\Delta x = \epsilon_x \times x$$

$$= 0.65 \times 10^{-3} \times 300 \times 10^{-3} = 0.195 \text{ mm}$$

$$x' = x + \Delta x$$



$$E = 70 \text{ GPa} \quad \delta \nu = 1/3$$

$$\epsilon_x = \frac{\sigma_x}{E} - \nu \frac{\sigma_z}{E}$$

$$= \frac{1}{E} \left(85 - \frac{1}{3} \times 180 \right)$$

$$= \frac{1}{70 \times 10^9} \times \frac{50}{10^3} \times 35 \times 10^6$$

$$\epsilon_x = 0.5 \times 10^{-3} \text{ m}$$

$$= 0.5 \text{ mm}$$

Date _____
Page _____

$$\sigma_x = 85 \text{ MPa}$$

$$\sigma_2 = 150 \text{ MPa}$$

$$AB = ?$$

$$D = ?$$

$$uee = ?$$

$$uee = ?$$

$$\epsilon_2 = \frac{\sigma_2}{E} - v \left(\frac{\sigma_x}{E} \right)$$

$$= \frac{150}{E} - \frac{1}{3} \times \frac{85}{E} \quad 28.33$$

$$= \frac{1}{E} (150 - 28.333)$$

$$= 121.667$$

$$70 \times 10^3$$

$$= 1.73 \text{ mm}$$

$$\epsilon_{xy} = -v \left(\frac{\sigma_x}{E} + \frac{\sigma_2}{E} \right)$$

$$= -\frac{1}{3} \times (28.33)$$

$$70 \times 10^3$$

$$= -1.119 \text{ mm}$$

$$\text{Volume of Plate} = ((1+\epsilon_x) 350)((1+\epsilon_2) 350) ((1+\epsilon_{xy}) 18)$$

(5) 12.5 mm thick
 $P = ?$ (Ans: horizontal displ. of AB)
 $q = 0.5 \text{ kPa}$

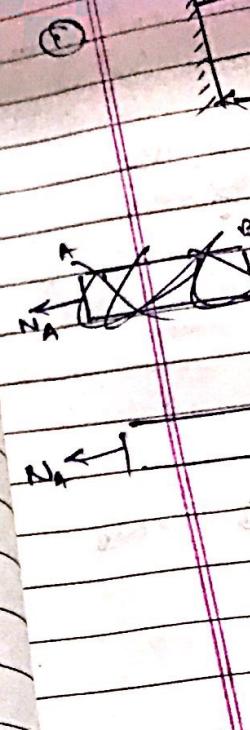
$$\text{Shear Stress} = \frac{F}{A} = \frac{P}{0.54}$$

$$\text{Shear Strain} = \frac{\text{Shear Stress}}{G} = \frac{4 \times 10^3}{0.6}$$

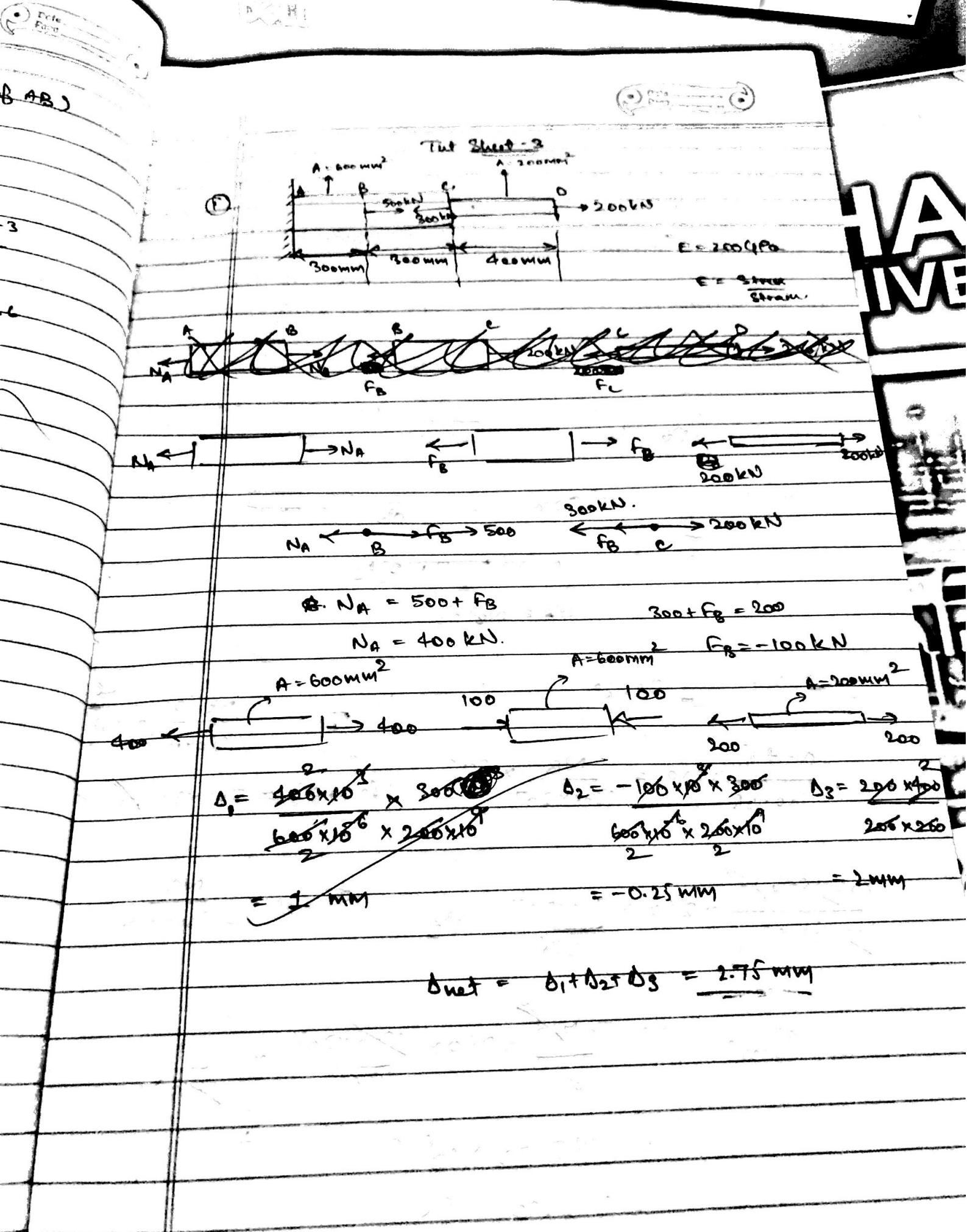
$$\eta = \frac{\text{Stress}}{\text{Strain}} = \frac{0.5 \times 10^9}{4 \times 10^{-3}} = \frac{P / 0.54}{0.6}$$

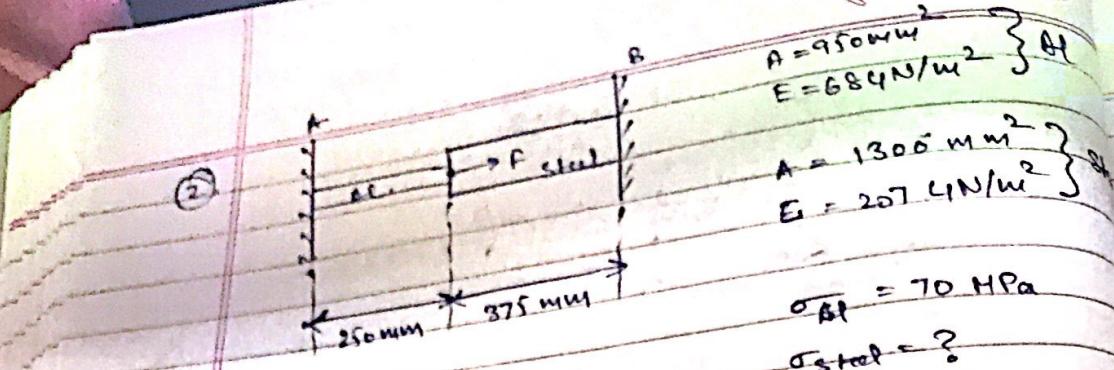
$$0.9 \times 0.5 \times 10^9 \times 4 \times 10^{-3} = P$$

$$1.8 \times 10^6 \text{ N} = P$$



During this very auspicious period you will be living through as





$$\Delta AP \approx \sigma_{Steel}$$

$$\frac{F}{950 \times 68} \times 250 = \frac{1300 \times 207}{1300 \times 207} \times STS$$



$$N_A = N_B + F$$

$$\Delta AP = \sigma_{Steel}$$

$$\sigma_{AP} = \frac{N_B + F}{950 \times 10^{-6}} = 70$$

$$\frac{N_B + F}{950 \times 68} \times 250 = \frac{N_B}{1300 \times 207} \times 375$$

$$(N_B + F) = 70 \times 950 \times 10^{-6}$$

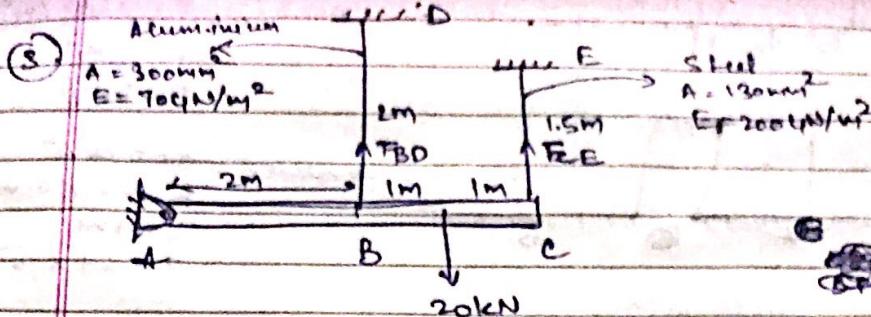
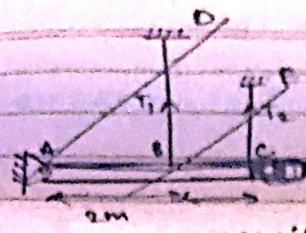
$$\frac{70 \times 950 \times 10^{-6}}{950 \times 68} \times 250 \times \frac{1300 \times 207}{375} = N_B$$

$$\sigma_{Steel} = \frac{N_B}{1300 \times 10^{-6}}$$

$$\sigma_{Steel} = \frac{70 \times 10^{-6}}{68} \times 250 \times \frac{1300 \times 207}{375} \times \frac{1}{1300 \times 10^{-6}}$$

$$= 142.05 \times 10^{-6}$$

$$= 142.05 \text{ MPa}$$



$$\text{Moment abt. A: } 2F_{BD} + 4F_{EE} = 3(20) = 60 \text{ kNm}$$

~~$$\frac{\Delta BD}{2} = \frac{\Delta CE}{\frac{A}{2}}$$~~

~~$$\Delta CE = 2 \Delta BD$$~~

$$70 \times 10^9 = \frac{F_{BD}}{300 \times 10^{-6}} \times \frac{2}{\Delta BD}$$

$$200 \times 10^9 = \frac{F_{EE}}{130 \times 10^{-6}} \times \frac{2}{\Delta CE}$$

~~$$3\Delta CE = 60 - 2F_{BD}$$~~

~~$$3F_{EE} = 60000 - 2F_{BD}$$~~

$$200 \times 10^9 \times 130 \times 10^{-6} \times (\frac{1}{2} \Delta CE) = 3F_{EE}$$

$$70 \times 10^9 \times 300 \times 10^{-6} \times \Delta BD = 2F_{BD}$$

$$(200 \times 130 \times 10^3 \times 4) + (70 \times 10^3 \times 300) \Delta BD = 60000$$

$$\Delta BD = 4.8 \times 10^{-4} \text{ m}$$

$$\Delta BD = 0.48 \text{ mm}$$

$$\Delta CE = 0.96 \text{ mm}$$

$$3F_{EE} = 200 \times 10^9 \times 130 \times 2 (0.96) \times 10^{-3}$$

$$2F_{BD} = 70 \times 10^9 \times 300 \times 0.48 \times 10^{-6}$$

$$F_{EE} = 16640 \text{ N}$$

$$F_{BD} = 5040 \text{ N}$$

$$\sigma_{stress} = 16.8 \text{ MPa}$$

$$\begin{aligned} \text{Stress} &= \frac{16640}{130 \times 10^{-6}} \\ &= 128 \text{ MPa} \end{aligned}$$

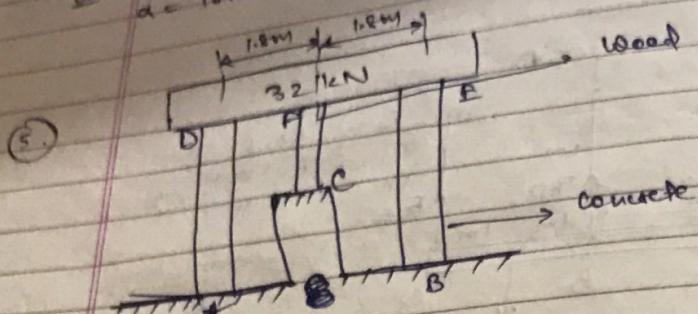
④ Copper rod
12mm dia
400 mm length
Heated from 20°C to 60°C
in Alum. tube ext Ø 20 mm
thickness 4 mm
same length

$$\text{At } E = 70 \text{ GPa}$$

$$\alpha = 23.7 \times 10^{-6} / ^\circ\text{C}$$

$$\text{At } E = 150 \text{ GPa}$$

$$\alpha = 18 \times 10^{-6} / ^\circ\text{C}$$



$$A = 0.04 \text{ m}^2$$

$$E = 20 \text{ GPa}$$

$$L = 2 \text{ m}$$

$$\alpha = 6 \times 10^{-6} / ^\circ\text{C}$$

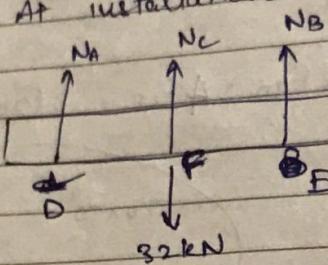
$$A = 0.08 \text{ m}^2$$

$$E = 20 \text{ GPa}$$

$$L = 4 \text{ m}$$

$$\alpha = 24 \times 10^{-6} / ^\circ\text{C}$$

a) At installation.



$$N_A = N_B$$

$$N_A + N_B + N_C = 32 \text{ kN}$$

$$N_B + 2N_A = 32 \text{ kN}$$

Moment abt A

$$1.8 N_C + 3.6 N_B = 1.8 (32)$$

$$N_C + 2N_B = 32$$

$$N_C = 32 - 2N_A$$

$$\frac{3}{5} N_A = 32 - 2N_A$$

$$3N_A = 160 - 10N_A$$

$$N_A = \frac{160}{13} = 12.30 \text{ kN}$$

$$N_C = 7.38 \text{ kN}$$

$$\sigma_C = \frac{7.38 \text{ kN}}{0.04 \text{ m}^2}$$

$$= 184.61 \text{ kPa}$$



(b) After a temp. decrease of -20°C

$$N_c + 3N_A = 32 \text{ kN} = 32000 \text{ N}$$

Also, $\Delta F = 540$

$$\frac{N_c \times 2 \times 10^3}{12 \times 0.04} + 2 \times 6 \times 10^3 \times 20 = \frac{N_A \times 4 \times 10^3}{20 \times 0.08} + 4 \times 24 \times 20$$

$$\frac{N_c \times 2 \times 10^3}{12 \times 0.04} + 12 \times 20 = \frac{N_A \times 4 \times 10^3}{20 \times 0.08} + 4 \times 24 \times 20$$

$$\frac{N_c - N_A}{12 \times 0.04} = \frac{2 \times N_A \times 10^3}{20 \times 0.08} = 24 \times 4 \times 20 - 12 \times 20$$

$$\frac{2 \times 10^3}{0.04} \left(\frac{N_c}{12} - \frac{N_A}{20} \right) = 84 \times 20 \times 10$$

~~$$\frac{N_c}{12} - \frac{N_A}{20} = \frac{84 \times 10 \times 0.04 \times 10^3}{20 \times 0.08}$$~~

$$5N_c - 3N_A = 84 \times 0.04 \times 10^4$$

~~$$5N_c - 3N_A = 20.16 \times 10^5$$~~

$$N_c = 32000 - 2N_A$$

$$5N_c = 20.16 \times 10^5 + 3N_A = 5(32000 - 2N_A)$$

~~$$20.16 \times 10^5 + 3N_A = 160000 - 10N_A$$~~

$$N_c = 32000 + (142769.23) \times 2$$

$$13N_A = -1856000$$

$$\sigma_c = \frac{N_c}{0.04} = 7938461.5 \text{ N/m}^2 \quad N_A = -142769.23 \text{ N}$$

$$= 7.938 \text{ MPa}$$

$$\sigma_A = \frac{N_A}{0.08} = -1784615.38 \text{ N}$$

$$= -1.784 \text{ MPa}$$

E_{AP} = 67.5 GPaDate _____
Page _____

$$\begin{aligned} \text{flange} \\ \text{ext } \phi &= 22 \text{ mm} \\ \text{ext } \phi &= 44 \text{ mm} \end{aligned}$$

$$\Delta l = \frac{1}{3} (24) \text{ mm} = 8 \text{ mm}$$

Stress in tube & bolt?

(b) Belt
350 mm long
80 mm wide
24 mm pitch
E_{steel} = 207 GPa

$$\sigma_{\text{steel}} + \sigma_{\text{AP}} = 0.8 \times 10^3$$

$$\frac{F \times 0.33}{\pi \times (10 \times 10^{-3})^2 \times 207 \times 10^9} + \frac{F \times 0.33}{\pi ((22 - 11)^2 \times 10^{-6}) \times 67.5}$$

this would give F.

$$\text{Now } \sigma_{\text{steel}} = \frac{F}{\pi \times (10 \times 10^{-3})^2}$$

$$\text{By } \sigma_{\text{flange}} = \frac{F}{\pi ((22 - 11)^2 \times 10^{-6})}$$

~~22/02/2017~~

$$\frac{T}{J} = \frac{I_{max}}{c} = \frac{G\phi}{L}$$

Tut - 4

- ① 5m long
stressed to 80 MPa $G = 83.4 \text{ GPa}$
shaft diameter = ?
Power at $\phi = 20^\circ$?

$$\frac{80 \times 10^6}{P} = \frac{83 \times 10^9 \times 4 \times \frac{\pi}{180} \times 10^3}{5}$$

$$P = \frac{80 \times 5 \times 180}{83 \times 4 \times \frac{\pi}{180} \times 10^3} = \frac{69.03}{10^3} = 0.069 \text{ m} \\ = 69 \text{ mm.}$$

$$P = 2\pi f T = 10T$$

$$\frac{T}{J} = \frac{\gamma}{P} = \frac{G\phi}{L}$$

$$T = \frac{83 \times 10^9 \times 4 \times \frac{\pi}{180}}{5} \times \frac{\pi}{2} \times (0.069)^4$$

$$T = 4.1337 \times 10^{-5+9} \text{ Nm}$$

$$P = 2 \times \pi \times 20 \times 4.1337 \times 10^{-4} \\ = 5194614.79 \text{ W} \\ = 5194 \text{ kW.}$$

$$② \phi_{max} = 3^\circ$$

$$L = 6 \text{ m}$$

$$T = 12 \text{ kN-m}$$

$$I_{max} = ?$$

$$G = 83.4 \text{ GPa}$$

$$I_{max} = \frac{83 \times 10^9 \times 3 \frac{\pi}{180} \times 0.075}{6} = 41.2 \text{ MPa}^4$$

$$\frac{T}{J} = \frac{G\phi}{L}$$

$$J = \frac{\pi}{2} c^3 = \frac{TL}{4\phi}$$

solving
 $d = 13.98 \text{ mm}$

$$41.2 \text{ MPa}^4 = \frac{12 \times 10^6 \times 6 \times 10^{-2}}{83 \times 10^9 \times 3 \frac{\pi}{180}} \times \frac{2}{\text{m}}$$

$$\textcircled{3} \quad P_{\text{avg}} = 100 \text{ kW}$$

$$\omega = 250 \text{ rpm} = \frac{250 \times 2\pi}{60} \text{ rad/sec}$$

$$\tau_{\text{max}} = 75 \text{ MPa}$$

$$\tau_{\text{max}} = P_{\text{avg}} (1.3) = 1.3 \times 100 \text{ kW} = 130 \text{ kW}$$

$$\frac{T}{J} = \frac{\tau_{\text{max}}}{c}$$

d = ?

$$P = T \cdot \omega$$

$$T = P/\omega = \frac{130 \times 10^3}{250 \times 2\pi} \times 60 = 4965.63 \text{ N}$$

$$\omega = 2\pi f$$

$$f = \frac{\omega}{2\pi}$$

$$\frac{T}{J} = \frac{\tau_{\text{max}}}{c}$$

$$\frac{4965.63}{\frac{\pi}{2} c^3} = \frac{75 \times 10^6}{c}$$

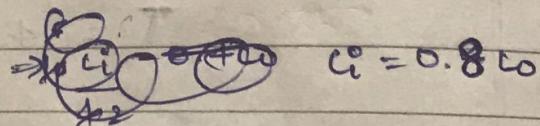
$$c^3 = \frac{4965.63}{\frac{\pi}{2} \times 75} \times 10^{-6}$$

$$c = 0.0348 \text{ m}$$

$$\text{a.) } d = 0.0696 \text{ m} = 69.6 \text{ mm}$$

b) hollow shaft

$$d_i = 0.8 d_o$$



$$c_i = 0.8 c_o$$

$$\frac{T}{J} = \frac{\tau_{\text{max}}}{c_o}$$

$$\frac{4965.63}{\frac{\pi}{2} (c_o^4 - c_i^4)} = \frac{75 \times 10^6}{c_o} = \frac{4965.63}{\frac{\pi}{2} (c_o)^4 (1 - 0.8^4)}$$

$$\frac{75 \times 10^6}{c_o} = 4965.63$$

$$\frac{\pi}{2} c_o^3 \times 0.1714 = 0.5904$$

$$d_o = 0.0702 \text{ m}$$

$$= 70.20 \text{ mm}$$

$$d_i = 56.16 \text{ mm}$$

$$c_o^3 = \frac{4965.63}{\frac{\pi}{2} \times 75 \times 10^6 \times 0.1714}$$

$$= 71.89 \times 10^{-6}$$

$$= 71.89 \times 10^{-6}$$

$$= 71.89 \times 10^{-6}$$

$$d_o =$$

$$d_i$$

$$\Rightarrow c$$

happen and clear these doubts in order to allow you starting real love relationship with them. After this is going for and this is going to happen very naturally during this very auspicious period you will be more than as

DELL

$\frac{T}{S} = \frac{\tau_{\text{max}}}{C}$
Date _____
Page _____
 $d = ?$
4965.63 N.H.

$$\frac{d_o^2 - d_i^2}{d_o^2} \times 100 = \text{Savings}$$

$$\frac{(70.20)^2 - (56.16)^2}{(69.6)^2} \times 100 = 36.62\%$$

$$\% \text{ Savings} = 100 - 36.62 = 63.37\%$$

HAPPY UNIVERS

$$\frac{\text{Area More} - \text{Area Less}}{\text{Area Less}} \times 100$$

$$\frac{(82.96)^2 - (66.368)^2}{(69.6)^2} \times 100$$

$$\frac{(\text{Solid}) \text{ Area} - \text{Hollow Area}}{\text{Solid Area}}$$

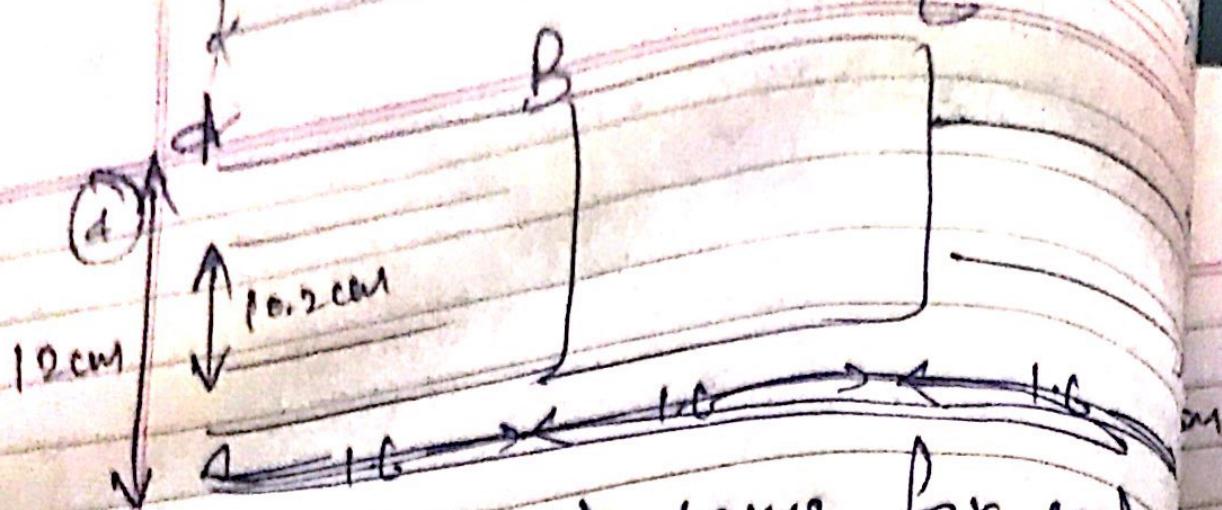
$$\% \text{ Saved} = \frac{(69.6)^2 - ((82.96)^2 - (66.368)^2)}{(66.368)^2} \times 100$$

$$= 48.85\%$$

$$d_o = 82.96 \text{ mm}$$

$$d_i = 0.8 \times d_o = 66.368$$

$$\Rightarrow C = 0.04148 \text{ mm}$$



Angle of twist same for each
length of each?

$$\frac{T_{max}}{C} = \frac{T}{I_p} = \frac{G\theta}{L}$$

Applied torque : ?

Total angle of twist

$$T_{max} = 50 \text{ MPa}$$

$$G = 84 \text{ GPa}$$

$$\frac{T_{max}}{C} = \frac{T}{I_p}$$

$$\frac{50 \times 10^6}{0.12} = \frac{T_1}{\frac{\pi}{2} (0.12)^4 \cdot (0.102)^4}$$

$$\frac{50 \times 10^6}{0.12} = \frac{T_2}{\frac{\pi}{2} (0.12)^4}$$

$$T_1 = 0.06 \times 10^6 \text{ Nm}$$

$$T_2 = 0.1357 \times 10^6 \text{ Nm}$$

~~$$\frac{50 \times 10^6}{0.096} = \frac{T_3}{\frac{\pi}{2} (0.096)^4}$$~~

$$T_3 = 7.539 \times 10^6 \text{ Nm}$$

$$\frac{\gamma_{max}}{c} = \frac{C_T \cdot e}{L}$$

Date _____
Page _____

$$\bullet \frac{c_1}{l_1} = \frac{c_2}{l_2} = \frac{c_3}{l_3}$$

Since same material
same would be
assume for all 3

Also given angle
of twist same

$$\frac{l_2}{l_1} = \frac{l_2}{l_2} = \frac{9.6}{l_3} \Rightarrow k$$

$$l_1 = \frac{12}{k} \quad l_2 = \frac{12}{k} \quad l_3 = \frac{9.6}{k}$$

$$l_1 + l_2 + l_3 = 480 \text{ cm}$$

$$\frac{1}{k} (12 + 12 + 9.6) = 480$$

$$k = \frac{33.6}{480} = 0.07$$

$$l_1 = \frac{12}{0.07} = 171.42 \text{ cm}$$

$$= 1.71 \text{ m}$$

$$l_2 = 1.71 \text{ m}$$

$$l_3 = 1.37 \text{ m}$$

$$\frac{\gamma_{max}}{c} = \frac{T}{I_p}$$

$$\frac{50 \times 10^6}{0.12} = \frac{T}{\frac{\pi}{2} ((0.12)^4 - (0.102)^4)}$$

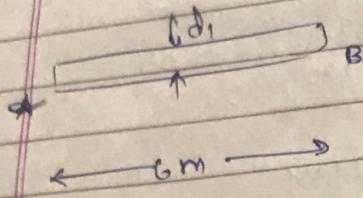
$$\Rightarrow T = 0.06 \times 10^6 \text{ Nm}$$

$$\frac{\gamma_{max}}{c} = \frac{C_T \theta}{L}$$

$$\frac{50 \times 10^6}{0.12} = \frac{84 \times 10^9 \times 0.10^3}{1.71} \Rightarrow \theta = 8.482 \times 10^{-6} \text{ rad} \\ = 0.48^\circ \\ 8\theta = 1.45^\circ$$

(a) $P = 100 \text{ kW}$
 40 rpm
 $\frac{2 \pi \times 90}{3600} \text{ rad/sec}$ ~~rad/sec~~ $\rightarrow \frac{4\pi}{3} \text{ rad/sec} = \omega$

$$\begin{aligned} P &= T \omega \\ 100 \times 10^3 &= T \frac{4\pi}{3} \\ T &= \frac{100 \times 10^3}{4\pi} \times 3 \\ T_B &= 2.387 \times 10^6 \text{ Nm} \end{aligned}$$



$$32 \times 10^3 \times \frac{3}{4\pi} = T_{AB}$$

$$T_{AB} = 7639.43 \text{ Nm}$$

$$T_{BC} = 16233.80 \text{ Nm}$$

$$\frac{\tau_{max}}{c} = \frac{T}{J} = \frac{4\theta}{L}$$

$$\begin{aligned} \tau_{max} &= 50 \text{ MPa} \\ c &= 80 \text{ GPa} \end{aligned}$$

$$\frac{50 \times 10^6}{d_1/2} = \frac{7639.43}{\frac{\pi}{2} \left(\frac{d_1}{2}\right)^4} = \frac{80 \times 10^9 \times \theta_1}{6}$$

$$\left(\frac{d_1}{2}\right)^3 = \frac{7639.43}{\pi/2 \times 50 \times 10^6}$$

$$\left(\frac{d_1}{2}\right)^3 = (9.726) \times 10^{-5}$$

$$\Rightarrow d_1 = 0.09198 \text{ m}$$

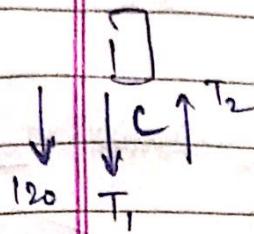
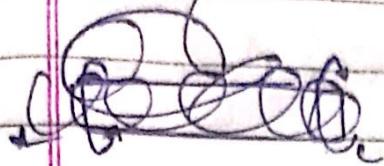
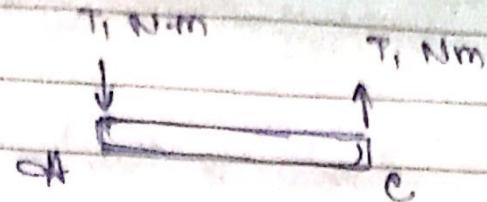
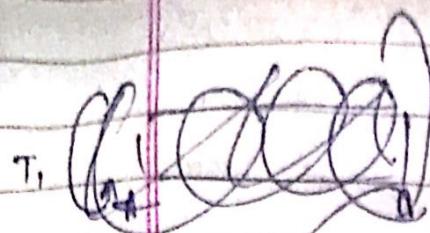
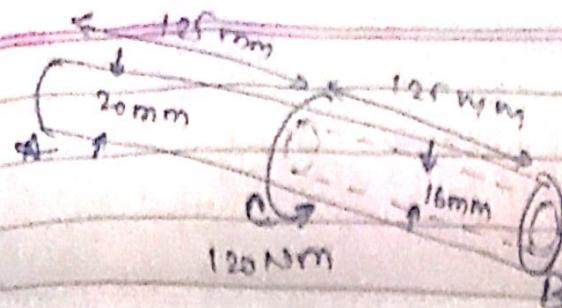
$$\frac{7639.43}{\frac{\pi}{2} \left(\frac{d_1}{2}\right)^4} = \frac{80 \times 10^9}{6} \times \theta_1$$

$$\theta_1 = 0.0815 \text{ rad}$$

$$= 4.67^\circ$$

Similarly get $d_2 \& \theta_2$.

(6)



Since both sides attached
to fixed support

$$\Theta_1 = \Theta_2$$

$$120 + T_1 = T_2$$

$$T_2 - T_1 = 120 \text{ Nm} \quad \frac{T}{J} = \frac{4\theta}{L}$$

$$\frac{TL}{J} = 4\theta = k$$

$$\frac{T_1 L_1}{J_1} = \frac{T_2 L_2}{J_2} = C$$

$$T_1 \neq T_2 ?$$

$$T_1 = \frac{C J_1}{L_1}$$

$$T_2 = \frac{C J_2}{L_2}$$

$$T_1 = C \frac{\frac{\pi}{2} (20 \times 10^{-3})^4}{125 \times 10^{-3}}$$

$$T_2 = C \frac{\frac{\pi}{2} ((10 \times 10)^{-3} - (8 \times 10)^{-3})^4}{125 \times 10^{-3}}$$

$$\frac{T}{J} = \frac{G\theta}{L} = \frac{\tau_{max}}{c}$$

Steel Shaft & Aluminium Tube
 $\tau_{max} = ?$ (Max. Torque)

$$\tau_{max} (\text{Steel}) = 120 \text{ MPa}; G = 80 \text{ GPa}$$

$$\tau_{max} (\text{Al}) = 70 \text{ MPa}; G = 27 \text{ GPa}$$

Angle of twist (θ) would be same
for both.

? 1-Hence would the net torque be divided
onto the shaft & alum. tube.

$$\frac{T}{J} = \frac{G\theta}{L}$$

$$\frac{T_1}{\frac{\pi}{2} \times (50 \times 10)^3 / 2} = \frac{120 \times 10^9}{50 \times 10^9}$$

$$\frac{TL}{GJ} = \theta$$

$$T_1 = \frac{\pi}{2} \times (50 \times 10)^3 \times 120 \times 10^9$$

$$\frac{T_1}{G_1 J_1} = \frac{T_2}{G_2 J_2}$$

$$= \frac{\pi}{2} \times 50 \times 120 \times 10^9$$

$$\frac{T_1}{G_1 J_1} = \frac{T_2}{G_2 J_2}$$

$$2945.24 \frac{23561.94 \text{ Nm}}{8}$$

$$\frac{T_1}{T_2} = \frac{G_1 J_1}{G_2 J_2}$$

$$T_2 = \frac{T_1 G_2 J_2}{G_1 J_1}$$

38
20
x

$$T = \frac{\pi}{2} \times 30 \times 10^{-3} \times 10^{-9} = \frac{1}{38 \times 10^9}$$

$$T = 61.559983 \text{ Nm}$$

$$= T_1 \times 27 \times \frac{1}{80} \times \frac{[(76 \times 10)^4 - (60 \times 10)^4]}{\frac{\pi}{2} \times 25 \times \left(\frac{50 \times 10^3}{2}\right)^4}$$

$$= 994.019 \times \left[\dots \right]$$

Possible

$$T_2 = 3244.825 \text{ Nm}$$

$$(28^4 \cdot 30^4) \times 10^{-10} \quad \frac{10 \times 10}{38 \times 10^{28}} \cdot 10$$

$$T_2 = 3689.69 \text{ Nm}$$

corresponding T_1 is

$$T_1 = \frac{C_1 J_1}{C_2 J_2} \cdot T_2$$

$$= \frac{80}{27} \times \frac{(25)^4}{\frac{38^4 \cdot 30^4}{8}} \times 3689.69$$
$$= 3349.04 \text{ Nm}$$

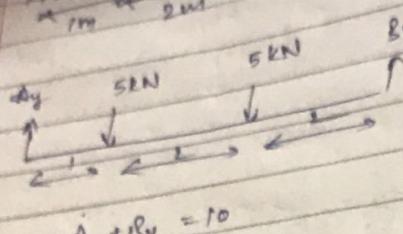
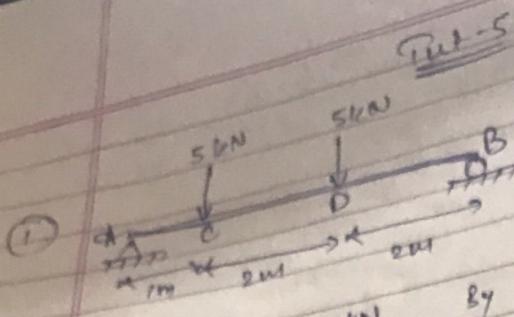
Since greater than the limit got from T_{max} of steel
so Not possible.

$$\Rightarrow T_1 = 2945.24 \text{ Nm}$$

$$T_2 = 3244.825 \text{ Nm}$$

$$T_{net} = T_1 + T_2$$

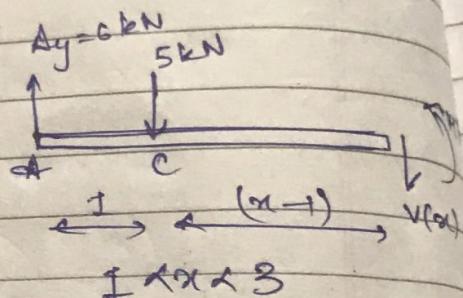
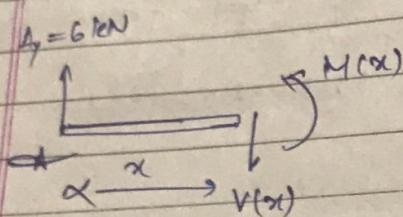
$$= 6190.065 \text{ Nm}$$



$$S(1) + S(3) = S'(B_y)$$

$$B_y = 4 \text{ kN}$$

$$\Rightarrow A_y = 6 \text{ kN}$$



$$0 \leq x < 1$$

$$G = 5 + V(x)$$

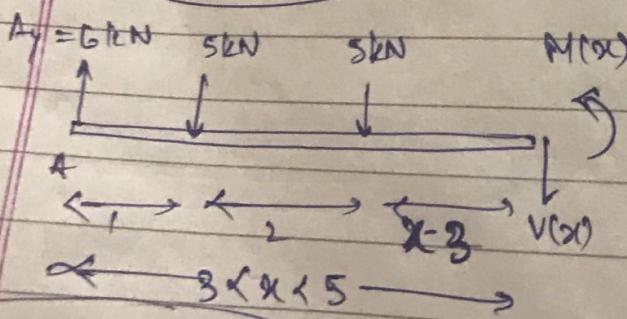
$$V(x) = 6 \text{ kN}$$

$$V(x) = 1 \text{ kN}$$

$$6x = M(x)$$

(F)

$$G(x) - 5(x-1) - M(x) = 0$$

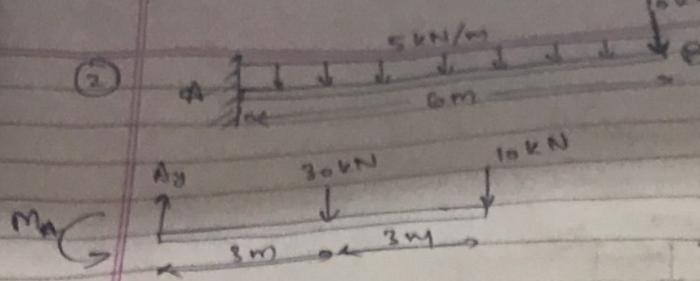


$$M(x) = x + 5$$

$$V(x) = -4 \text{ kN}$$

$$G(x) - 5(x-1) - 5(x-3) - M(x) = 0$$

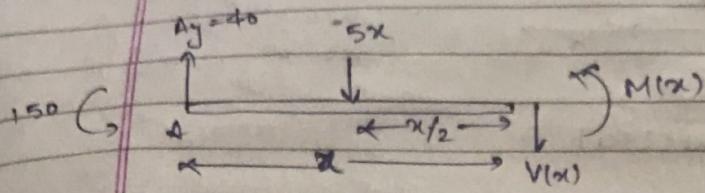
$$M(x) = -4x + 20$$



$$A_y = 40 \text{ kN}$$

$$30(3) + 10(6) = M_A$$

$$M_A = 150 \text{ kNm}$$

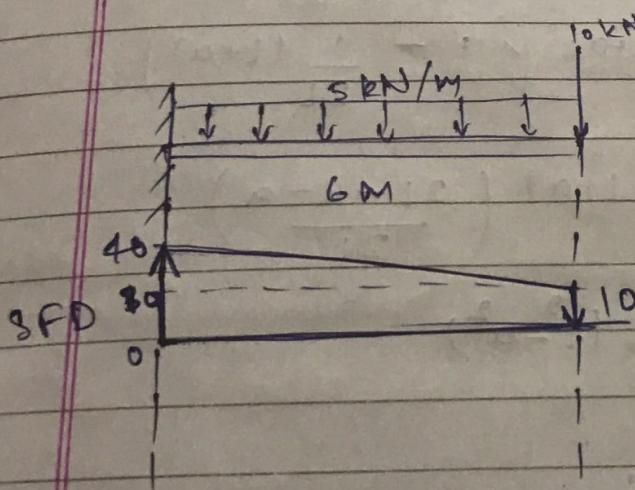


$$0 < x < 6$$

$$40 - 5(x) = V(x)$$

$$-150 + 40(x) - 5(x)\left(\frac{x}{2}\right) - M(x) = 0$$

$$M(x) = -\frac{5x^2}{2} + 40x - 150$$



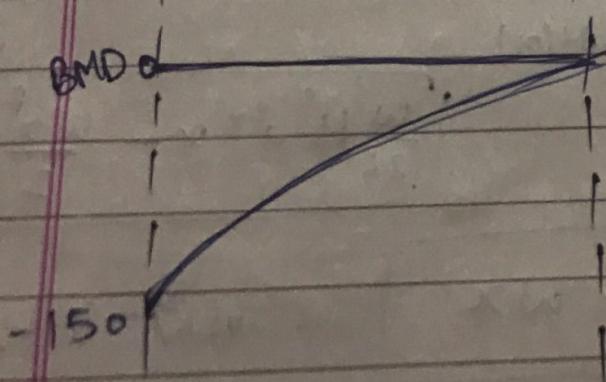
$$M(0) = -150$$

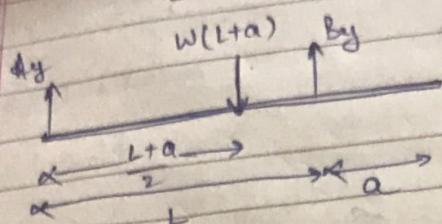
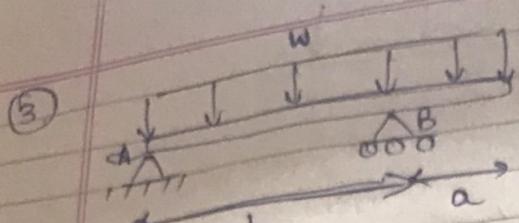
$$M(3) = -22.5 + 120 - 150 \\ = -52.5$$

$$M(6) = -90 + 240 - 150 \\ = 0$$

$$M'(x) = -5x + 40$$

Max/Min at
 $x = 8$





$$Ay + By = w(L+a)$$

$$\frac{w(L+a)^2}{2} = By(L)$$

$$By = \frac{w}{2L} (L+a)^2$$

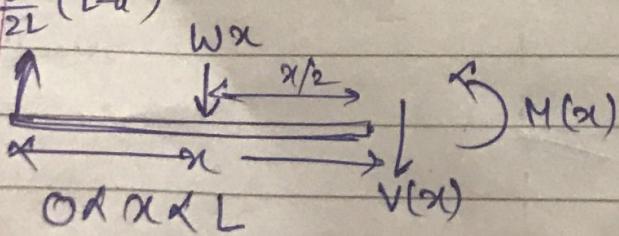
$$Ay = w(L+a) - \frac{w}{2L} (L+a)^2$$

$$= w(L+a) \left(1 - \frac{1}{2L} (L+a) \right)$$

$$= w(L+a) \left(\frac{2L - L-a}{2L} \right)$$

$$Ay = \frac{w}{2L} (L-a^2)$$

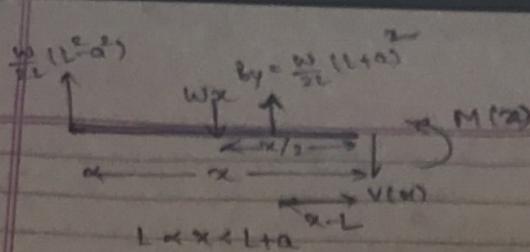
$$Ay = \frac{w}{2L} (L-a^2)$$



$$\boxed{\frac{w}{2L} (L-a^2)x - \frac{wx^2}{2} = M(x)}$$

$$\boxed{V(x) = \frac{w}{2L} (L-a^2) - wx}$$

$$V(L) = \frac{wL}{2} - \frac{wa^2}{2L} - w \\ = -WL - \frac{wa^2}{2L}$$

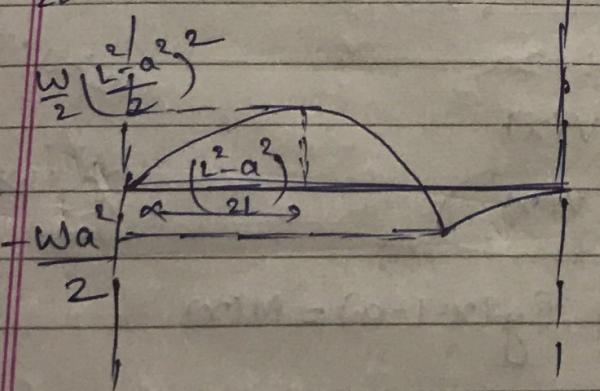
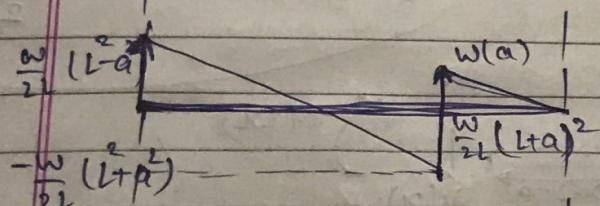
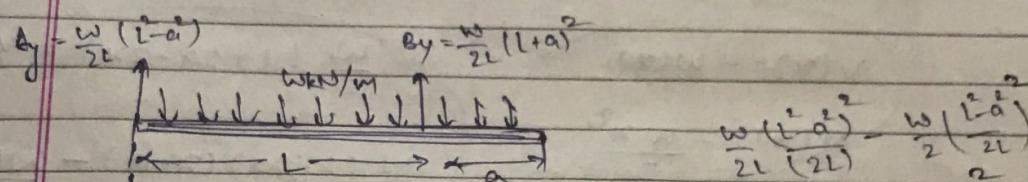


$$\frac{w}{2L} (L^2 - a^2 + L^2 + a^2 + 2La) - w(x) = V(x)$$

$$\frac{w}{2L} w(L^2 + 2La) - w(x) = V(x)$$

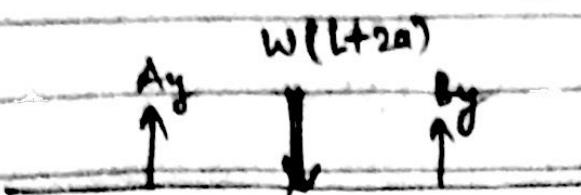
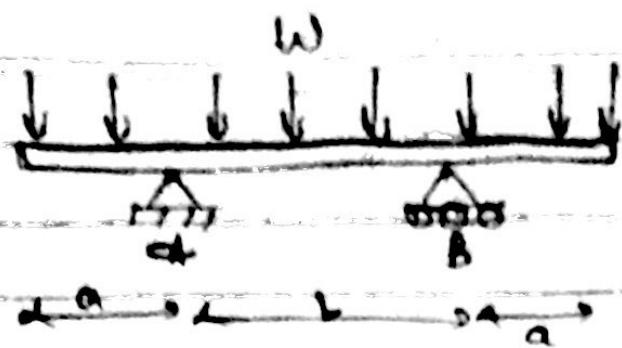
$$w(L+a-x) = V(x) \rightarrow w(L+a) - w(x)$$

$$\frac{w}{2L} (L^2 - a^2)(x) - wx\left(\frac{x}{2}\right) + \frac{w}{2L} (L+a)^2(x-L) = M(x)$$

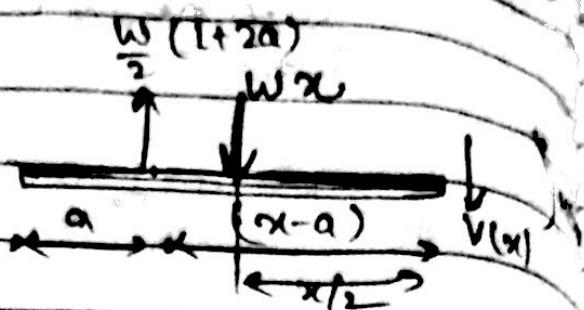
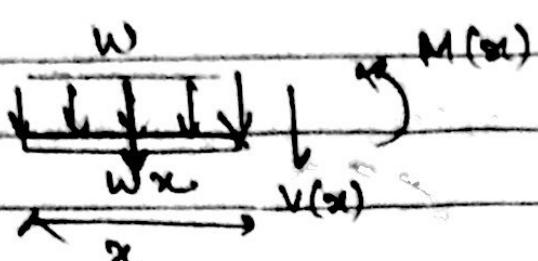


$$\frac{w}{2L} (L-a)^2 (L+a)^2 - \frac{w}{2} (L+a)^2 + \frac{w}{2L} (L+a)^2 (a)$$

$$\frac{w(L+a)^2}{2} \left(\frac{L-a}{OL} - 1 + \frac{a}{L} \right) = 0$$



$$A_y = B_y = \frac{w}{2}(L+2a)$$



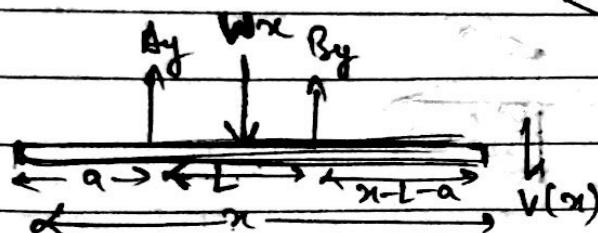
$$0 < x < a$$

$$V(x) = -w(x)$$

$$M(x) = -\frac{w x^2}{2}$$

$$\frac{w}{2}(L+2a-2x) = V(x)$$

$$\frac{w}{2}(L+2a)(x-a) - \frac{w x^2}{2} = M(x)$$



$$L+a < x < L+2a$$

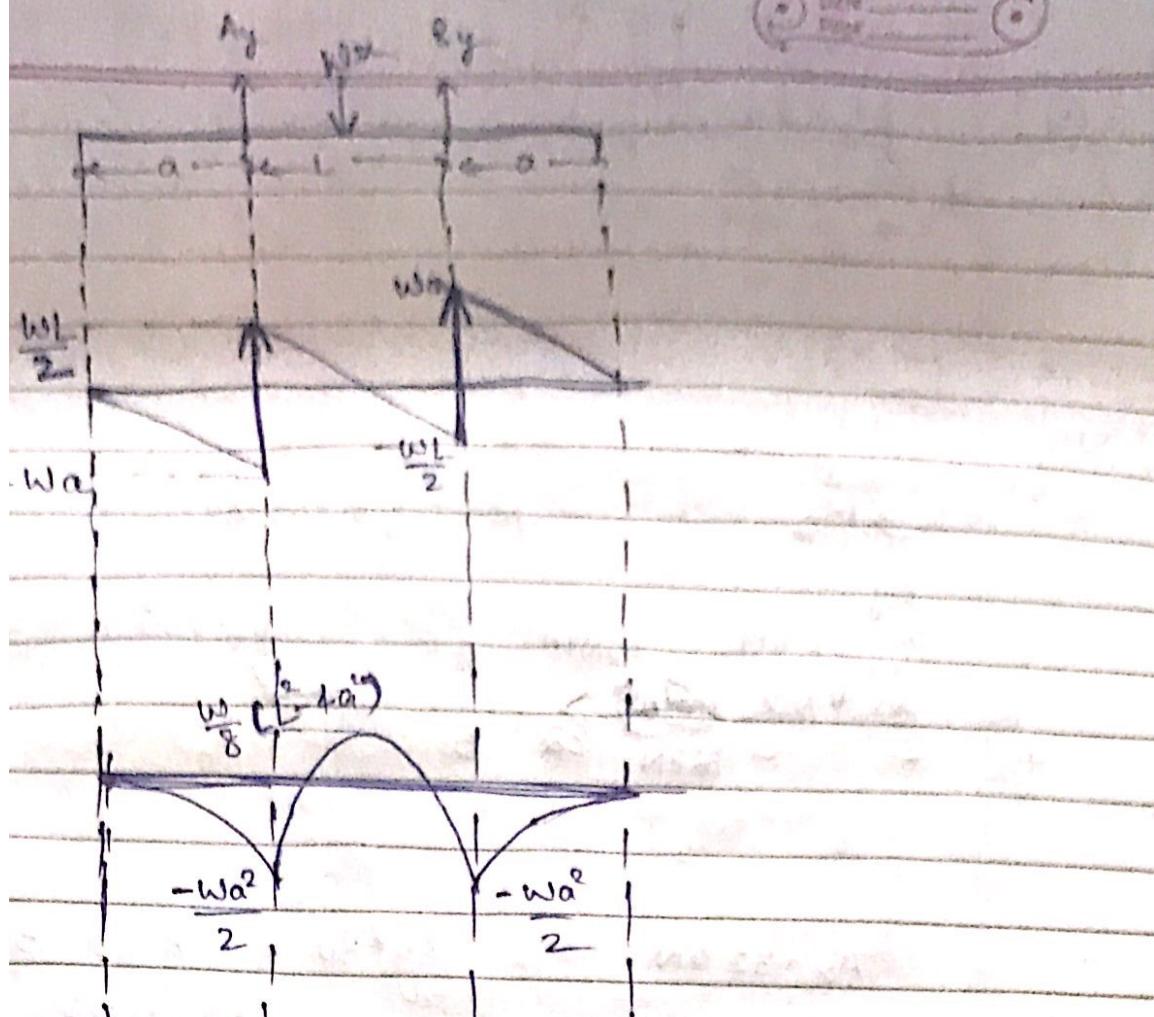
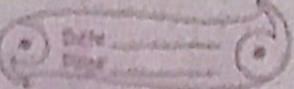
$$A_y + B_y - w x = V(x)$$

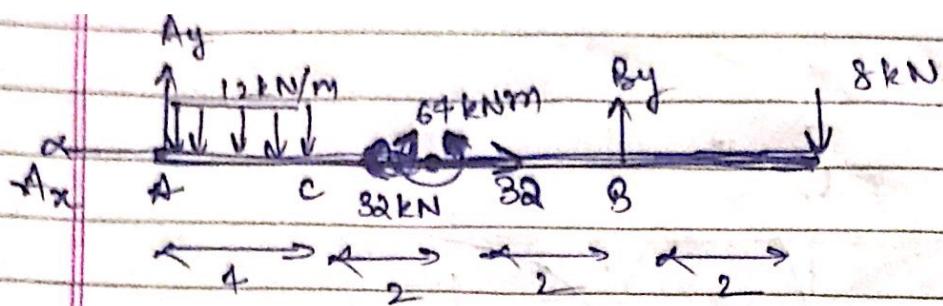
$$w(L+2a-x) = V(x)$$

$$\frac{w x(L+2a-x)}{2} - \frac{w x^2}{2}$$

$$\frac{w x^2}{2} + w x a - \frac{w x^2}{2} = \frac{w a^2}{2} + w x a$$

$$A_y(x-a) - \frac{w x^2}{2} + B_y(x-1-a) = M(x)$$





$$Ax = 32 \text{ kN}$$

$$Ay + By = 48 + 8 = 56$$

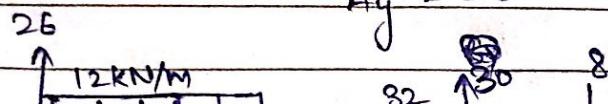
$$Ay + By = 56$$

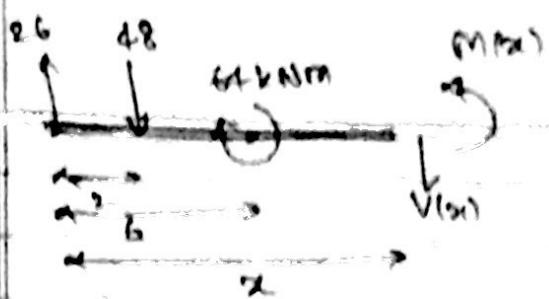
f_e

$$48(2) + 64 + 8(10) = By (8)$$

$$\Rightarrow By = 30$$

$$Ay = 26$$



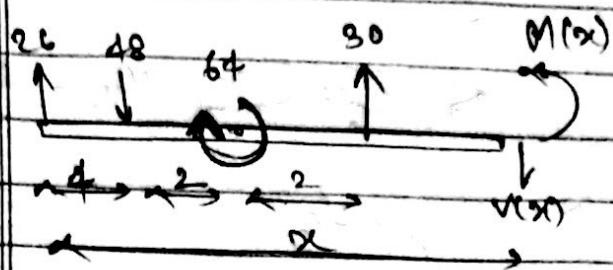


$$64 \times 2 = 8$$

$$V(x) = -22$$

$$26(x) - 48(x-2) + 64 = M(x)$$

$$M(x) = -22x + 160$$



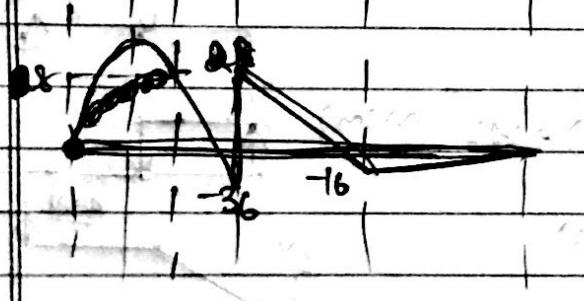
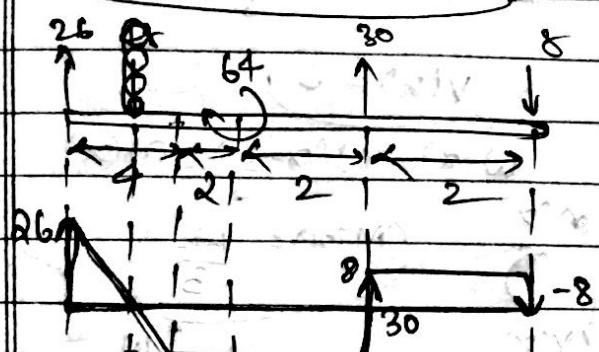
$$8 \times 2 = 20$$

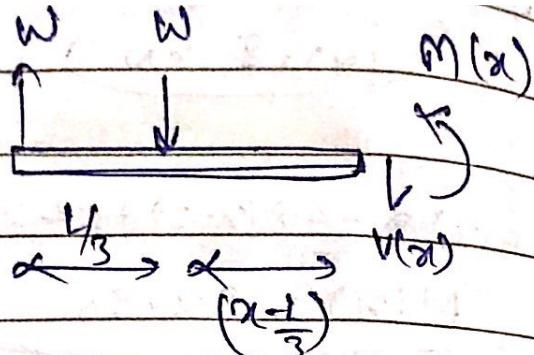
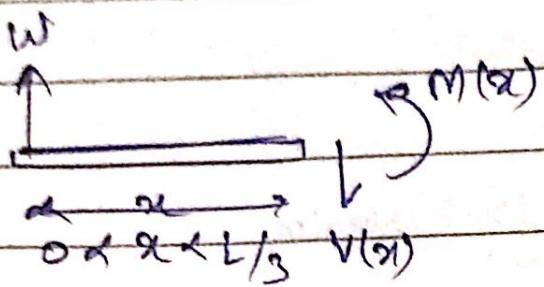
$$V(x) = 26 + 30 - 48 = 8$$

$$V(x) = 8 \text{ kN}$$

$$26(x) - 48(x-2) + 64 + 30(x-8) = M(x)$$

$$M(x) = 8x - 80$$





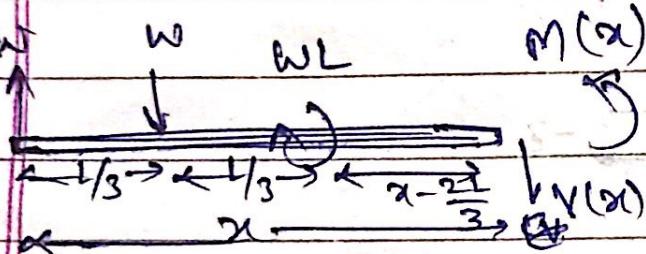
8.

$$V(x) = W$$

$$M(x) = Wx$$

$$V(x) = 0$$

$$W(x) - W\left(x - \frac{L}{3}\right) = M(x)$$

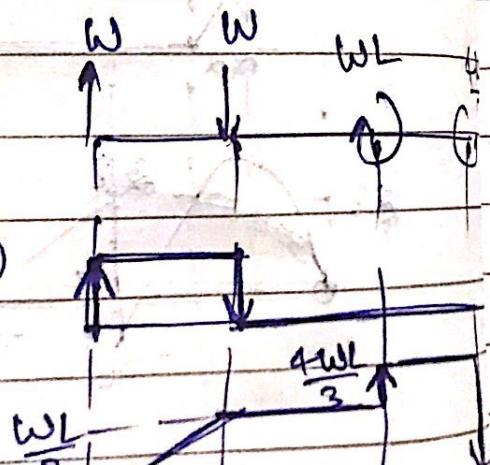


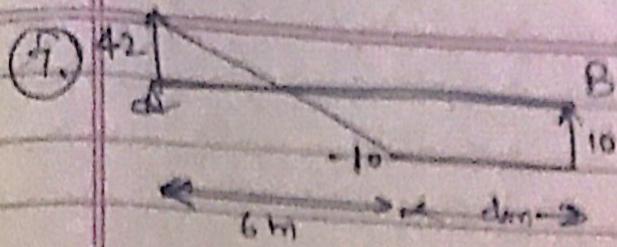
$$M(x) = \frac{WL}{3}$$

$$V(x) = 0$$

~~$$W(x) - W\left(x - \frac{L}{3}\right) + WL = M(x)$$~~

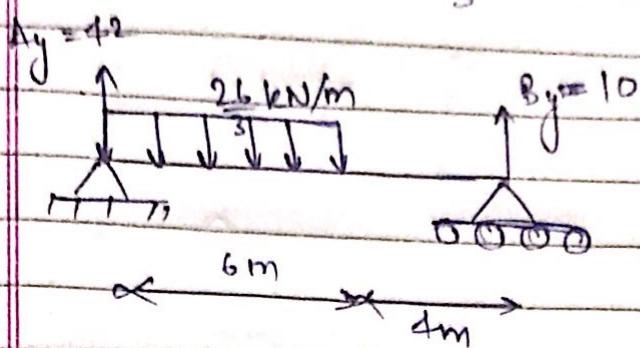
$$\frac{4WL}{3} = M(x)$$



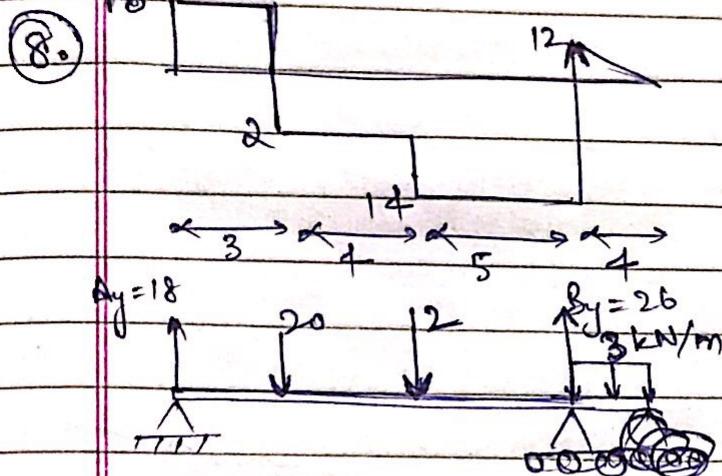


$$-10 = m(B) + 42$$

$$\frac{-16}{3} = m = -\frac{16}{3}$$



18



$$0 = m(4) + 12$$

$$m = -3$$

$$y = -3x + 12$$