## 2016

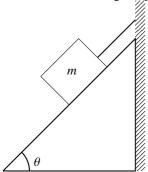
## ME: Mechanical Engineering

## AI24BTECH11022 - Pabbuleti Venkata Charan Teja

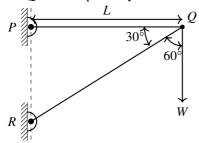
40) A block of mass *m* rests on an inclined plane and is attached by a string to the wall as shown in the figure. The coefficient of static friction between the plane and the block is 0.25. The string can withstand a maximum force of 20*N*. The maximum value of the mass (*m*) for which the string will not break and the block will be in static equilibrium is \_\_\_\_\_ *kg*.

Take  $\cos \theta = 0.8$  and  $\sin \theta = 0.6$ .

Acceleration due to gravity  $g = 10m/s^2$ 

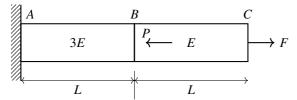


41) A two-member truss PQR is supporting a load W. The axial forces in members PQ and QR are respectively



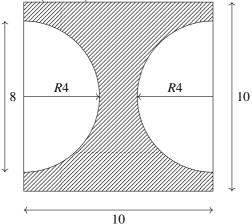
- a) 2W tensile and  $\sqrt{3}W$  compressive
- b)  $\sqrt{3}W$  tensile and 2W compressive
- c)  $\sqrt{3}W$  compressive and 2W tensile
- d) 2W compressive and  $\sqrt{3}W$  tensile
- 42) A horizontal bar with a constant cross-section is subjected to loading as shown in the figure. The Young's moduli for the sections *AB* and *BC* are 3*E* and *E*, respectively.

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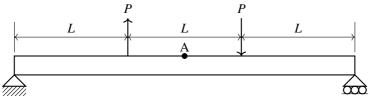
For the deflection at C to be zero, the ratio P/F is \_\_\_\_\_

43) The figure shows cross-section of a beam subjected to bending. The area moment of inertia (in  $mm^4$ ) of this cross-section about its base is \_\_\_\_\_



All dimensions are in mm

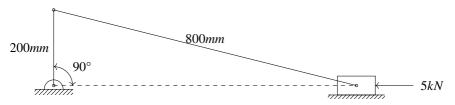
44) A simply-supported beam of length 3L is subjected to the loading shown in the figure.



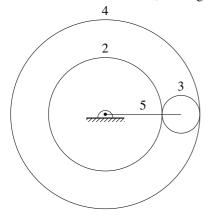
It is given that P = 1N, L = 1m and Young's modulus E = 200GPa. The cross-section is a square with dimension  $10mm \times 10mm$ . The bending stress (in Pa) at the point A located at the top surface of the beam at a distance of 1.5L from the left end is \_\_\_\_\_

(Indicate compressive stress by a negative sign and tensile stress by a positive sign)

45) A slider crank mechanism with crank radius 200mm and connecting rod length 800mm is shown. The crank is rotating at 600rpm in the counterclockwise direction. In the configuration shown, the crank makes an angle of  $90^{\circ}$  with the sliding direction of the slider, and a force of 5kN is acting on the slider. Neglecting the inertia forces, the turning moment on the crank (in KN-m) is \_\_\_\_\_

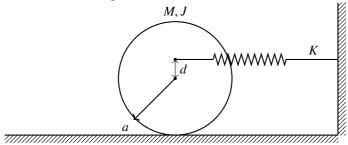


46) In the gear train shown, gear 3 is carried on arm 5. Gear 3 meshes with gear 2 and gear 4. The number of teeth on gear 2, 3 and 4 are 60, 20 and 100 respectively. If gear 2 is fixed and gear 4 rotates with an angular velocity of 100*rpm* in the counterclockwise direction, the angular speed of arm 5 (in *rpm*) is



- a) 166.7 counterclockwise
- b) 166.7 clockwise

- c) 62.5 counterclockwise
- d) 62.5 clockwise
- 47) A solid disc with radius a is connected to a spring at a point d above the center of the disc. The other end of the spring is fixed to the vertical wall. The disc is free to roll without slipping on the ground. The mass of the disc is M and the spring constant is K. The polar moment of inertia for the disc about its centre is  $J = \frac{Ma^2}{2}$ .



The natural frequency of this system in rad/s is given by

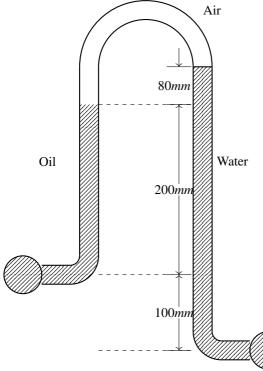
a) 
$$\sqrt{\frac{2K(a+d)^2}{3Ma^2}}$$

c) 
$$\sqrt{\frac{2K(a+d)^2}{Ma^2}}$$

- 48) The principal stresses at a point inside a solid object are  $\sigma_1 = 100MPa$ ,  $\sigma_2 =$ 100MPa and  $\sigma_3 = 0MPa$ . The yield strength of the material is 200MPa. The factor of safety calculated using Tresca (maximum shear stress) theory is  $n_T$  and the factor of safety calculated using von Mises (maximum distortional energy) theory is  $n_V$ . Which one of the following relations is TRUE?
  - a)  $n_T = \left(\frac{\sqrt{3}}{2}\right) n_V$ b)  $n_T = \left(\sqrt{3}\right) n_V$

- c)  $n_T = n_V$ d)  $n_V = (\sqrt{3}) n_T$
- 49) An inverted U-tube manometer is used to measure the pressure difference between two pipes A and B, as shown in the figure. Pipe A is carrying oil (specific gravity = 0.8) and pipe B is carrying water. The densities of air and water are and  $1.16kg/m^3$  and  $1000kg/m^3$ , respectively. The pressure difference between pipes A and B is \_ kPa.

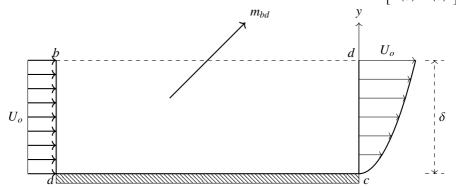
Acceleration due to gravity  $g = 10m/s^2$ .



50) Oil (kinematic viscosity,  $v_{oil} = 1.0 \times 10^{-5} m^2/s$ ) flows through a pipe of 0.5m diameter with a velocity of 10m/s. Water (kinematic viscosity,  $v_w = 0.89 \times 10^{-6} m^2/s$ ) is flowing

through a model pipe of diameter 20mm. For satisfying the dynamic similarity, the velocity of water (in m/s) is \_\_\_\_\_

51) A steady laminar boundary layer is formed over a flat plate as shown in the figure. The free stream velocity of the fluid is  $U_o$ . The velocity profile at the inlet a - b is uniform, while that at a downstream location c - d is given by  $u = U_o \left[ 2 \left( \frac{y}{\delta} \right) - \left( \frac{y}{\delta} \right)^2 \right]$ .



The ratio of the mass flow rate,  $m_{bd}$ , leaving through the horizontal section b-d to that entering through the vertical section a-b is \_\_\_\_\_

52) A steel ball of 10mm diameter at 1000K is required to be cooled to 350K by immersing it in a water environment at 300K. The convective heat transfer coefficient is  $1000W/m^2 - K$ . Thermal conductivity of steel is 40W/m - K. The time constant for the cooling process  $\tau$  is 16s. The time required (in s) to reach the final temperature is \_\_\_\_\_