

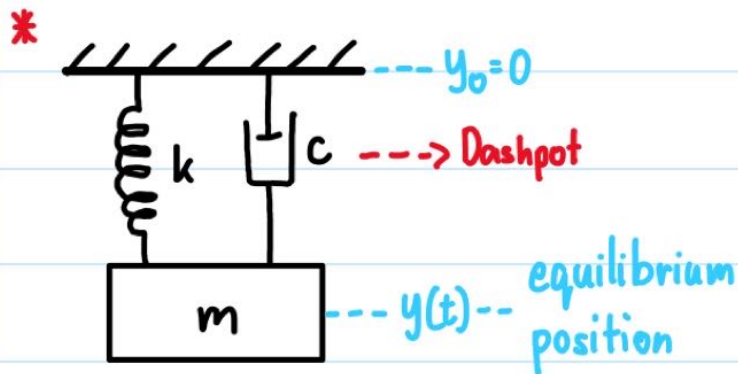
VIBRATION ENGINEERING

LONG EXAM 2 NOTES

One Degree of Freedom System with Viscous Damping

Tuesday, January 30, 2024

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* where:

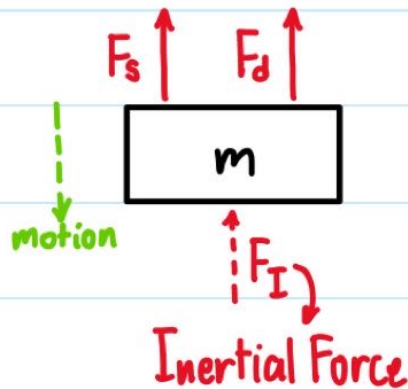
k = spring constant

c = damping coefficient

m = mass

~ occurs only for a system with a dashpot.

~ use Newton's 2nd Law of Motion:



* F_s = force due to spring
 $= k(y - y_0) = ky$

F_d = force due to damping
 $= c(\dot{y} - \dot{y}_0) = c\dot{y}$

$F_I = ma = m\ddot{y}$

$$\sum F_y = 0$$

$$F_I + F_d + F_s = 0$$

$$m\ddot{y} + c\dot{y} + ky = 0 \quad \textcircled{1} \text{ Differential Eq. of Motion}$$

~ evaluating eq. $\textcircled{1}$:

$$[m\ddot{y} + c\dot{y} + ky = 0] \frac{1}{m}$$

$$\ddot{y} + \frac{c}{m}\dot{y} + \frac{k}{m}y = 0 \quad \textcircled{2}$$

* where:

Natural Frequency
 $\omega_n^2 = \frac{k}{m} \quad \textcircled{3}$

$$\zeta = \frac{c}{2m\omega_n} \quad \textcircled{4}$$

"zeta" = viscous damping factor

~ combining eqns. ②, ③, & ④:

$$\ddot{y} + 2\zeta\omega_n\dot{y} + \omega_n^2 y = 0$$

Differential Eq. of Motion
w/ Zeta

~ the solution yields:

$$y(t) = A_1 e^{s_1 t} + A_2 e^{s_2 t} \quad \text{⑤}$$

~ where A is a constant and S is a quantity to be determined.

$$s^2 + 2\zeta\omega_n s + \omega_n^2 = 0$$

~ use quadratic formula:

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = \frac{-2\zeta\omega_n \pm \sqrt{4\zeta^2\omega_n^2 - 4(1)\omega_n^2}}{2(1)}$$

$$= \frac{-\cancel{2}\zeta\omega_n \pm \cancel{2}\omega_n\sqrt{\zeta^2 - 1}}{\cancel{2}}$$

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = (-\zeta \pm \sqrt{\zeta^2 - 1})\omega_n \quad \text{⑥}$$

~ combining ⑤ & ⑥: (eliminating s_1 & s_2)

$$y(t) = A_1 e^{(-\zeta + \sqrt{\zeta^2 - 1})\omega_n t} + A_2 e^{(-\zeta - \sqrt{\zeta^2 - 1})\omega_n t}$$

$$y(t) = e^{-\zeta\omega_n t} \left(A_1 e^{\sqrt{\zeta^2 - 1}\omega_n t} + A_2 e^{-\sqrt{\zeta^2 - 1}\omega_n t} \right)$$

$$* \omega_d = \omega_n \sqrt{\zeta^2 - 1}$$

* NOTE:

- ~ if $\zeta < 1$, then: under damping case
- ~ if $\zeta = 1$, then: critical damping case
- ~ if $\zeta > 1$, then: overdetermined case

[OR]

~ use L.E.M.: (let $q_i = y$)

$$\frac{\partial}{\partial t} \left[\frac{\partial T}{\partial \dot{y}} \right] - \frac{\partial T}{\partial y} + \frac{\partial V}{\partial y} = Q_i \quad \text{---} \rightarrow \text{forcing function} \quad \textcircled{a}$$

* not sure here

$$\textcircled{b} \quad * Q_i = F_i - \frac{\partial \mathcal{F}}{\partial \dot{y}} ; \mathcal{F} = \frac{1}{2} \sum c_n \delta \dot{y}^2 \quad \textcircled{c}$$

* where:

$$T = \frac{1}{2} m \dot{y}^2$$

$$V = \frac{1}{2} k (y - \cancel{y_0})^2 = \frac{1}{2} k y^2$$

$$\mathcal{F} = \frac{1}{2} c (\dot{y} - \cancel{\dot{y}_0})^2 = \frac{1}{2} c \dot{y}^2$$

$$\frac{\partial}{\partial t} \left[\frac{\partial T}{\partial \dot{y}} \right] = \frac{\partial}{\partial t} \left[\frac{1}{2} (2) m \dot{y} \right] = m \ddot{y}$$

$$\frac{\partial V}{\partial y} = \frac{1}{2} (2) k y = k y$$

$$\frac{\partial \mathcal{F}}{\partial \dot{y}} = \frac{1}{2} (2) c \dot{y} = c \dot{y}$$

$$\frac{\partial T}{\partial y} = 0, F_i = 0$$

~ thus,

$$m\ddot{y} - 0 + ky = -c\dot{y}$$
$$\left[m\ddot{y} + c\dot{y} + ky = 0 \right] \frac{1}{m}$$

$$\ddot{y} + \frac{c}{m}\dot{y} + \frac{k}{m}y = 0 \quad (1)$$

~ let, $\omega_n^2 = \frac{k}{m}$ (2a), $\zeta = \rho =$ viscous damping factor

$$\frac{c \cdot (2\omega_n)}{m \cdot (2\omega_n)} = \frac{2\omega_n c}{2m\omega_n} \quad (2b)$$

~ thus,

$$\underline{\ddot{y} + 2\rho\omega_n\dot{y} + \omega_n^2 y = 0} \quad (3)$$

~ the complementary function:

$$\underline{y(t) = A_1 e^{s_1 t} + A_2 e^{s_2 t}} \quad (4)$$

~ where A is a constant and S is a quantity to be determined.

$$\underline{s^2 + 2\rho\omega_n s + \omega_n^2 = 0} \quad (5)$$

~ use quadratic formula:

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = \frac{-2\rho\omega_n \pm \sqrt{4\rho^2\omega_n^2 - 4(1)\omega_n^2}}{2(1)}$$

$$= \frac{-\cancel{2}\rho\omega_n \pm \cancel{2}\omega_n\sqrt{\rho^2 - 1}}{\cancel{2}}$$

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = (-\rho \pm \sqrt{\rho^2 - 1})\omega_n \quad (6)$$

~ combining ④ & ⑥: (eliminating s_1 & s_2)

$$y(t) = A_1 e^{(-\zeta + \sqrt{\zeta^2 - 1})\omega_n t} + A_2 e^{(-\zeta - \sqrt{\zeta^2 - 1})\omega_n t}$$

$$y(t) = e^{-\zeta\omega_n t} \left(A_1 e^{\sqrt{\zeta^2 - 1}\omega_n t} + A_2 e^{-\sqrt{\zeta^2 - 1}\omega_n t} \right)$$

$$* \omega_d = \omega_n \sqrt{\zeta^2 - 1}$$

General Solution for Single

⑦ Degree of Freedom System

w/ Viscous Damping

3 Cases:

1) if $\zeta < 1$, then: under damping case

2) if $\zeta = 1$, then: critical damping case

3) if $\zeta > 1$, then: overdetermined case

3 Cases of Viscous Damping

Thursday, February 22, 2024

8:21 AM

CASE 1: Underdamped Free Vibration (if $\rho < 1$)

Analytical Method:

~ using eq. 6:

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = (-\rho \pm \sqrt{\rho^2 - 1}) \omega_n$$

* if $\rho < 1$,

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = -\rho \omega_n \pm \omega_n \sqrt{(1 - \rho^2)(-1)}$$

$$= -\rho \omega_n \pm \omega_n \sqrt{(1 - \rho^2)} i \quad (8)$$

~ thus,

$$y(t) = e^{-\rho \omega_n t} [C_1 \cos(\omega_n \sqrt{1 - \rho^2} t) + C_2 \sin(\omega_n \sqrt{1 - \rho^2} t)] \quad (9)$$

~ Differentiating eq. 9:

$$\dot{y}(t) = e^{-\rho \omega_n t} [C_1 \cos(\omega_n \sqrt{1 - \rho^2} t) + C_2 \sin(\omega_n \sqrt{1 - \rho^2} t)](-\rho \omega_n) + e^{-\rho \omega_n t} [-C_1 \sin(\omega_n \sqrt{1 - \rho^2} t) + C_2 \cos(\omega_n \sqrt{1 - \rho^2} t)](\omega_n \sqrt{1 - \rho^2}) \quad (10)$$

~ when $t = 0$,

$$\text{Eq. 9: } y(0) = e^0 [C_1 \cos^1(0) + C_2 \sin^0(0)] = y_0$$

$$y_0 = C_1 \quad (11)$$

$$\text{Eq. 10: } \dot{y}(0) = e^0 [C_1 \cos^1(0) + C_2 \sin^0(0)](-\rho \omega_n) + e^0 [-C_1 \sin^0(0) + C_2 \cos^1(0)](\omega_n \sqrt{1 - \rho^2}) = \dot{y}_0$$

$$\dot{y}_0 = -\rho \omega_n C_1 + \omega_n \sqrt{1 - \rho^2} C_2 \quad (12)$$

~ combine eq. 11 & 12:

$$\dot{y}_0 = -\zeta \omega_n y_0 + \omega_n \sqrt{1-\zeta^2} C_2$$

$$C_2 = \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_n \sqrt{1-\zeta^2}} \quad (13)$$

~ combine eq. 9, 11, & 13: (eliminating C_1 & C_2)

$$y(t) = e^{-\zeta \omega_n t} \left[y_0 \cos(\omega_n \sqrt{1-\zeta^2} t) + \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_n \sqrt{1-\zeta^2}} \sin(\omega_n \sqrt{1-\zeta^2} t) \right]$$

~ but,

$$\omega_d = \omega_n \sqrt{1-\zeta^2} \quad \text{Frequency of Damped Free Vibration}$$

* Displacement Response:

$$y(t) = e^{-\zeta \omega_n t} \left[y_0 \cos(\omega_d t) + \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_d} \sin(\omega_d t) \right]$$

* Velocity Response:

$$\begin{aligned} \dot{y}(t) = & e^{-\zeta \omega_n t} \left[y_0 \cos(\omega_d t) + \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_d} \sin(\omega_d t) \right] (-\zeta \omega_n) \\ & + e^{-\zeta \omega_n t} \left[-y_0 \sin(\omega_d t) + \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_d} \cos(\omega_d t) \right] \omega_d \end{aligned}$$

$$\therefore \dot{y}(t) = e^{-\zeta \omega_n t} \left[\left(y_0 \cos(\omega_d t) + \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_d} \sin(\omega_d t) \right) (-\zeta \omega_n) + \left(-y_0 \sin(\omega_d t) + \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_d} \cos(\omega_d t) \right) \omega_d \right]$$

~ evaluating eq. 9:

$$y(t) = e^{-\zeta \omega_n t} \left[C_1 \cos(\omega_d t) + C_2 \sin(\omega_d t) \right]$$

* let, $C_1 = A \sin(\phi_d)$

$$C_2 = A \cos(\phi_d)$$

~ then:

$$y(t) = e^{-\zeta \omega_n t} [A \sin(\phi_d) \cos(\omega_d t) + A \cos(\phi_d) \sin(\omega_d t)]$$

$$y(t) = A e^{-\zeta \omega_n t} \sin(\omega_d t + \phi_d) \rightarrow \text{(Damped) Phase Angle}$$

Amplitude

Frequency of Damped Free Vibration

~ let us find A:

$$c_1^2 + c_2^2 = A^2 [\sin^2(\phi_d) + \cos^2(\phi_d)]$$

$$A = \sqrt{y_0^2 + \left(\frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_n \sqrt{1 - \zeta^2}} \right)^2}$$

$$\therefore y(t) = \sqrt{y_0^2 + \left(\frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_n \sqrt{1 - \zeta^2}} \right)^2} e^{-\zeta \omega_n t} \sin(\omega_d t + \phi_d)$$

RKN Method:

~ let: $k = 1.0$

when $t=0$:

$$c = 1.0$$

$$y(0) = y_0 = 2.0$$

$$m = 1.0$$

$$\dot{y}(0) = \dot{y}_0 = 1.0$$

$$h = 0.5$$

also: $\omega_n^2 = \frac{k}{m} = 1.0$

$$\zeta = \frac{c}{2m\omega_n} = \frac{1.0}{2(1)(1)} = 0.5$$

$$\therefore \zeta = 0.5 < 1$$

~ The Differential Equation of Motion:

$$m\ddot{y} + c\dot{y} + ky = 0$$

$$\ddot{y} + \frac{c}{m}\dot{y} + \frac{k}{m}y = 0$$

$$\ddot{y} + 2\beta\omega_n\dot{y} + \omega_n^2 y = 0 \quad (1)$$

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = \frac{-2\beta\omega_n \pm \sqrt{4\beta^2\omega_n^2 - 4(1)\omega_n^2}}{2(1)}$$

$$= \frac{-\cancel{2}\beta\omega_n \pm \cancel{2}\omega_n\sqrt{1-\beta^2}i}{\cancel{2}}$$

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = -\beta\omega_n \pm \omega_n\sqrt{1-\beta^2}i \quad (2)$$

~ then the complementary function:

$$y(t) = e^{-\beta\omega_n t} [C_1 \cos(\omega_n\sqrt{1-\beta^2}t) + C_2 \sin(\omega_n\sqrt{1-\beta^2}t)] \quad (3)$$

~ differentiating eqn. 3:

$$\begin{aligned} \dot{y}(t) = & e^{-\beta\omega_n t} [C_1 \cos(\omega_n\sqrt{1-\beta^2}t) + C_2 \sin(\omega_n\sqrt{1-\beta^2}t)](-\beta\omega_n) \\ & + e^{-\beta\omega_n t} [-C_1 \sin(\omega_n\sqrt{1-\beta^2}t) + C_2 \cos(\omega_n\sqrt{1-\beta^2}t)](\omega_n\sqrt{1-\beta^2}) \end{aligned}$$

* Recall:

$$C_1 = y_0 = 2.0$$

$$C_2 = \frac{\dot{y}_0 + y_0\beta\omega_n}{\omega_n\sqrt{1-\beta^2}} = \frac{1 + 2(0.5)(1)}{(1)\sqrt{1-(0.5)^2}} = \underline{2.3094}$$

~ use excel *

~ In RKN use 6 auxiliary eqns:

$$\bullet k_1 = \frac{1}{2}h \cdot f(t_n, y_n, \dot{y}_n)$$

$$\bullet k_2 = \frac{1}{2}h \cdot f(t_n + \frac{1}{2}h, y_n + K, \dot{y}_n + k_1); \quad K = \frac{1}{2}h(\dot{y}_n + \frac{1}{2}k_1)$$

$$\bullet k_3 = \frac{1}{2}h \cdot f(t_n + \frac{1}{2}h, y_n + K, \dot{y}_n + k_2)$$

$$\bullet k_4 = \frac{1}{2}h \cdot f(t_n + \frac{1}{2}h, y_n + L, \dot{y}_n + 2k_3); \quad L = h(\dot{y}_n + k_3)$$

- $y_{n+1} = y_n + h \left[\dot{y}_n + \frac{1}{3} (k_1 + k_2 + k_3) \right]$ ~ for displacement
- $\dot{y}_{n+1} = \dot{y}_n + \frac{1}{3} (k_1 + 2(k_2 + k_3) + k_4)$ ~ for velocity

~ using eq. 1:

$$\ddot{y} + 2\beta\omega_n \dot{y} + \omega_n^2 y = 0$$

$$\ddot{y} = -2\beta\omega_n \dot{y} - \omega_n^2 y = f(t_n, y_n, \dot{y}_n)$$

~ then when $n=0$,

$$k_1 = \frac{1}{2}h \cdot f(t_0, y_0, \dot{y}_0) = \frac{1}{2}h (-2\beta\omega_n \dot{y}_0 - \omega_n^2 y_0)$$

$$= \frac{1}{2}(0.5) (-2(0.5)(1)(1) - (1)^2(2)) = \underline{-0.75}$$

$$* K = \frac{1}{2}h (\dot{y}_0 + \frac{1}{2}k_1) = \frac{1}{2}(0.5) (1 + \frac{1}{2}(-0.75))$$

$$= \underline{0.15625}$$

$$k_2 = \frac{1}{2}h \cdot f(t_0 + \frac{1}{2}h, y_0 + K, \dot{y}_0 + k_1)$$

$$= \frac{1}{2}(0.5) (-2(0.5)(1)(1-0.75) - (1)^2(2+0.15625))$$

$$= \underline{-0.60156}$$

$$k_3 = \frac{1}{2}h \cdot f(t_0 + \frac{1}{2}h, y_0 + K, \dot{y}_0 + k_2)$$

$$= \frac{1}{2}(0.5) (-2(0.5)(1)(1-0.60156) - (1)^2(2+0.15625))$$

$$= \underline{-0.63867}$$

$$* L = h (\dot{y}_0 + k_3) = 0.5 (1 - 0.63867) = \underline{0.18066}$$

$$k_4 = \frac{1}{2}h \cdot f(t_0 + \frac{1}{2}h, y_0 + L, \dot{y}_0 + 2k_3)$$

$$= \frac{1}{2}(0.5) (-2(0.5)(1)(1+2(-0.63867)) - (1)^2(2+0.18066))$$

$$= \underline{-0.47583}$$

~ then at $n=1$,

$$y_1 = y_0 + h \left[\dot{y}_0 + \frac{1}{3} (k_1 + k_2 + k_3) \right]$$

$$= 2 + 0.5 \left[1 + \frac{1}{3} (-0.75 - 0.60156 - 0.63867) \right]$$

$$\boxed{y_1 = 2.18629 \text{ m}}$$

$$\dot{y}_1 = \dot{y}_0 + \frac{1}{3}(k_1 + 2(k_2 + k_3) + k_4)$$

$$= 1 + \frac{1}{3}(-0.75 + 2(-0.60156 - 0.63867) - 0.47583)$$

$$\dot{y}_1 = -0.23543 \text{ m/s}$$

* if you want to decrease ζ , increase mass.

if stepsize is decreased, accuracy increases & reduce errors.

CASE 2: Critically Damped Free Vibration ($\zeta = 1$)

$$\ddot{y} + 2\zeta\omega_n\dot{y} + \omega_n^2 y = 0 \quad (1)$$



$$s^2 + 2\zeta\omega_n s + \omega_n^2 = 0 \quad (2)$$

~ using quadratic formula:

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = -\zeta\omega_n \pm \omega_n\sqrt{\zeta^2 - 1} ; \text{ if } \zeta = 1 \quad (3)$$

$\underbrace{\hspace{10em}}_{=0}$

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = -\zeta\omega_n \text{ (repeated roots)} \quad (4)$$

~ the complementary function:

$$y(t) = e^{-\zeta\omega_n t} (C_1 + C_2 t) \quad (5) \text{ General Solution for Displacement}$$

~ differentiating eq. 5:

$$\dot{y}(t) = e^{-\zeta\omega_n t} (C_2) + (C_1 + C_2 t) e^{-\zeta\omega_n t} (-\zeta\omega_n)$$

$$\dot{y}(t) = e^{-\zeta\omega_n t} [-\zeta\omega_n C_1 + C_2(1 - \zeta\omega_n t)] \quad (6) \text{ General Solution for Velocity}$$

* When $t=0$, let initial values be: $y(0) = y_0$, $\dot{y}(0) = \dot{y}_0$

~ using eqn. 5:

$$y_0 = \cancel{e^0} [C_1 + \cancel{C_2(0)}]$$

$$\therefore \underline{C_1 = y_0} \quad (7)$$

~ using eqn. 6:

$$\dot{y}_0 = e^0 [-\zeta \omega_n y_0 + C_2 (1 - \zeta \omega_n (0))] \quad \text{⑧}$$

$$\therefore C_2 = \dot{y}_0 + y_0 \zeta \omega_n \quad \text{⑨}$$

~ then,

Displacement: $y(t) = e^{-\zeta \omega_n t} [y_0 + (\dot{y}_0 + y_0 \zeta \omega_n)t] \quad \text{⑩}$

Velocity: $\dot{y}(t) = e^{-\zeta \omega_n t} [-\zeta \omega_n y_0 + (\dot{y}_0 + y_0 \zeta \omega_n)(1 - \zeta \omega_n t)] \quad \text{⑪}$

Ex.

1.) let $m = 1.0$

$k = 1.0$

$C = 2.0$

$$\omega_n = \sqrt{\frac{k}{m}} = 1.0$$

$$\zeta = \frac{C}{2m\omega_n} = 1.0 \rightarrow (\zeta = 1)$$

\therefore Case 2

~ let $y_0 = 2.00$

$\dot{y}_0 = 1.00$

[excel demonstration]

~ analytical & RKN are the same

CASE 3: Overdamped Free Vibration ($\zeta > 1$)

$$\ddot{y} + 2\zeta\omega_n\dot{y} + \omega_n^2 y = 0$$



$$s^2 + 2\zeta\omega_n s + \omega_n^2 = 0$$

~ using quadratic formula:

$$\begin{bmatrix} s_1 \\ s_2 \end{bmatrix} = -\zeta\omega_n \pm \omega_n \sqrt{\zeta^2 - 1} ; \text{ if } \zeta > 1, \quad \text{⑫}$$

roots are

real & distinct

~ then the complementary function is:

$$y(t) = c_1 e^{[-j\omega_n + \omega_n \sqrt{\beta^2 - 1}]t} + c_2 e^{[-j\omega_n - \omega_n \sqrt{\beta^2 - 1}]t}$$

$$y(t) = e^{-j\omega_n t} [c_1 e^{\omega_n \sqrt{\beta^2 - 1} t} + c_2 e^{-\omega_n \sqrt{\beta^2 - 1} t}] \quad (12) \text{ for Displacement}$$

~ differentiating eq. 12:

$$\dot{y}(t) = e^{-j\omega_n t} [(\omega_n \sqrt{\beta^2 - 1}) c_1 e^{\omega_n \sqrt{\beta^2 - 1} t} + (-\omega_n \sqrt{\beta^2 - 1}) c_2 e^{-\omega_n \sqrt{\beta^2 - 1} t}] + e^{-j\omega_n t} (-j\omega_n) [c_1 e^{\omega_n \sqrt{\beta^2 - 1} t} + c_2 e^{-\omega_n \sqrt{\beta^2 - 1} t}]$$

$$\dot{y}(t) = e^{-j\omega_n t} (\omega_n \sqrt{\beta^2 - 1}) [c_1 e^{\omega_n \sqrt{\beta^2 - 1} t} - c_2 e^{-\omega_n \sqrt{\beta^2 - 1} t}] + e^{-j\omega_n t} (-j\omega_n) [c_1 e^{\omega_n \sqrt{\beta^2 - 1} t} + c_2 e^{-\omega_n \sqrt{\beta^2 - 1} t}] \quad (13) \text{ for velocity}$$

* When $t=0$, let initial values be: $y(0) = y_0$, $\dot{y}(0) = \dot{y}_0$

~ using eqn. 12:

$$y_0 = c_1 + c_2$$

$$y_0 = c_1 + c_2$$

$$\therefore c_1 = y_0 - c_2 \quad (14)$$

~ using eqn. 13:

$$\dot{y}_0 = (\omega_n \sqrt{\beta^2 - 1}) [c_1 - c_2] \quad (15)$$

$$+ (-j\omega_n) [c_1 + c_2] \quad * c_1 = y_0 - c_2, y_0 = c_1 + c_2$$

$$\dot{y}_0 = \omega_n \sqrt{\beta^2 - 1} (y_0 - c_2 - c_2) - j\omega_n y_0$$

$$c_2 = \frac{1}{2} \left[y_0 - \frac{\dot{y}_0 + y_0 j\omega_n}{\omega_n \sqrt{\beta^2 - 1}} \right] \quad (16)$$

~ then,

$$c_1 = y_0 - \frac{1}{2} \left[y_0 - \frac{\dot{y}_0 + y_0 j\omega_n}{\omega_n \sqrt{\beta^2 - 1}} \right] \rightarrow = y_0 - \frac{1}{2} y_0 + \frac{(1)}{2} \frac{\dot{y}_0 + y_0 j\omega_n}{\omega_n \sqrt{\beta^2 - 1}} = \frac{1}{2} y_0 + \frac{\dot{y}_0 + y_0 j\omega_n}{2\omega_n \sqrt{\beta^2 - 1}}$$

$$c_2 = \frac{1}{2} \left[y_0 - \frac{\dot{y}_0 + y_0 j\omega_n}{\omega_n \sqrt{\beta^2 - 1}} \right]$$

~ Finally,

Displacement

$$y(t) = e^{-\zeta \omega_n t} \left[\left(\frac{1}{2} y_0 + \frac{\dot{y}_0 + y_0 \zeta \omega_n}{2 \omega_n \sqrt{\zeta^2 - 1}} \right) e^{\omega_n \sqrt{\zeta^2 - 1} t} + \frac{1}{2} \left(y_0 - \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_n \sqrt{\zeta^2 - 1}} \right) e^{-\omega_n \sqrt{\zeta^2 - 1} t} \right]$$

(18)

Velocity

$$\dot{y}(t) = e^{-\zeta \omega_n t} (\omega_n \sqrt{\zeta^2 - 1}) \left[\left(\frac{1}{2} y_0 + \frac{\dot{y}_0 + y_0 \zeta \omega_n}{2 \omega_n \sqrt{\zeta^2 - 1}} \right) e^{\omega_n \sqrt{\zeta^2 - 1} t} - \frac{1}{2} \left(y_0 - \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_n \sqrt{\zeta^2 - 1}} \right) e^{-\omega_n \sqrt{\zeta^2 - 1} t} \right] + e^{-\zeta \omega_n t} (-\zeta \omega_n) \left[\left(\frac{1}{2} y_0 + \frac{\dot{y}_0 + y_0 \zeta \omega_n}{2 \omega_n \sqrt{\zeta^2 - 1}} \right) e^{\omega_n \sqrt{\zeta^2 - 1} t} + \frac{1}{2} \left(y_0 - \frac{\dot{y}_0 + y_0 \zeta \omega_n}{\omega_n \sqrt{\zeta^2 - 1}} \right) e^{-\omega_n \sqrt{\zeta^2 - 1} t} \right]$$

(19)

~ Plot & compare with RKN.