

# Analysis of Dynamical Systems

26/2/21

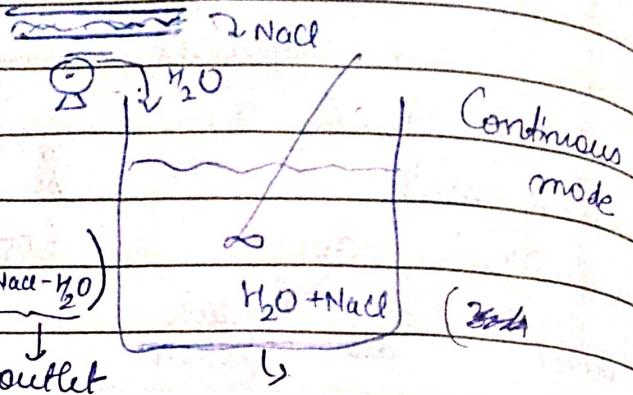
Date  
Page

## in Transform Domain

Input rate - output rate

= accumulation  
rate

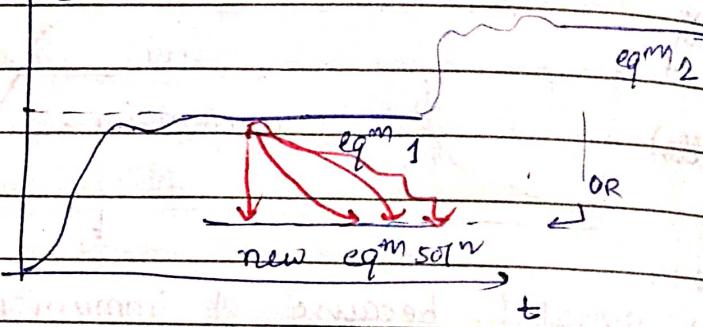
$$\Rightarrow \frac{d(c_{\text{NaCl}})}{dt} = f(g_{H_2O}, m_{\text{NaCl}}, Q_{\text{NaCl-H}_2O})$$



Determine  $\text{eq}^m \text{ sol}^n$  to the above  $\text{eq}^n$  - why needed?

Once we reach  $\text{eq}^m$ , ↑ e

fluctuations after ↓ SS,  
we will have reach new  
 $\text{eq}^m \text{ sol}^n$ .



Objective: How does the system evolve b/w 2  $\text{eq}^m$  solutions.

• Dynamics b/w 2  $\text{eq}^m \text{ sol}^n$  is Transfer Domain modeling:

First order dynamics -

$$a_1 \frac{dy}{dt} + a_0 y = b u(t) \quad \text{--- (1)}$$

$$\Rightarrow \frac{a_1}{a_0} \frac{dy}{dt} + y = \frac{b}{a_0} u(t)$$

Standard notation  $\Rightarrow I \frac{dy}{dt} + y = K u(t)$

$I$  input variable / forcing function

$$I = \frac{a_1}{a_0}$$

and

$$K = \frac{b}{a_0}$$

↑  
Time constant

Process gain / static gain / gain

Here  $y: c_{\text{NaCl}}$

$u: m_{\text{NaCl}}$  → forcing a change in system  $\Rightarrow$  output variable

system at steady state "when a disturbance is introduced" to the system ( $t=0$ )  
 This means, the system was at steady state before.  
 At that time, this eqn is valid:

$$t=0 \quad T \frac{dy_s}{dt} + y_s = k u_s(t) \quad \text{--- (3)}$$

Governing eqn is the eqn at 1st steady state.

(2) - (3)

$$T \frac{d(y-y_s)}{dt} + (y-y_s) = k(u-u_s(t))$$

Let  $y-y_s = y^*$  — deviation variable

$$T \frac{dy^*}{dt} + y^* = k u^*(t) \quad \text{--- (4)}$$

We can conveniently apply Laplace transform to eqn (4)

$$T s \bar{y}^*(s) - y^*(0) + \bar{y}^*(s) = k \bar{u}^*(s)$$

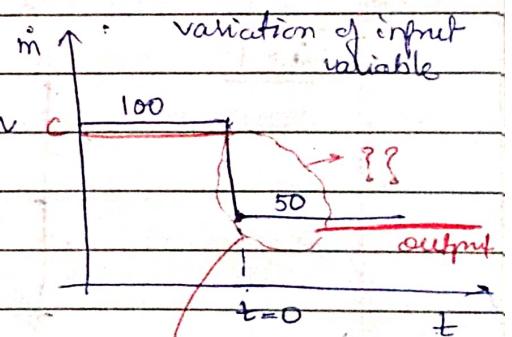
$\Rightarrow$  (so, the eqn 4 is convenient)

$$(Ts+1) \bar{y}^*(s) = k \bar{u}^*(s)$$

$$\bar{y}^*(s) = \frac{k \bar{u}^*(s)}{(Ts+1)} \Rightarrow \bar{y}^*(s) = \frac{k}{s} \frac{\bar{u}^*(s)}{(s+1)} = g(s)$$

Input Nacl in = 100 kg/h

suddenly the flow rate drops to 50 kg/h



This type of forcing function is called an

a) ideal step input

$$u(t) = \begin{cases} 0 & t < 0 \\ A & t > 0 \end{cases}$$

How does  $c$  vary

$$g(s) = \frac{k}{s+1} = \frac{\bar{y}^*(s)}{s \bar{u}^*(s)}$$

$$\Rightarrow \bar{y}^*(s) = \frac{k \cdot A}{s+1} ; \bar{u}^*(s) = \frac{A}{s}$$

$$\therefore \bar{y}^*(s) = \frac{AK}{s+1} + \frac{A}{s} \left[ -\frac{1}{s+1} + \frac{1}{s} \right] \leftarrow \text{partial fraction}$$

$$y^*(t) = AK \left[ \frac{1}{s} - \frac{1}{s + \frac{1}{\tau}} \right]$$

$$y^*(t) = AK \left[ 1 - e^{-t/\tau} \right]$$

*b = 2*

$$y(t) = b(1 - e^{-t/a})$$

*b = 1.5*

*b = 1*

Convention: input in blue

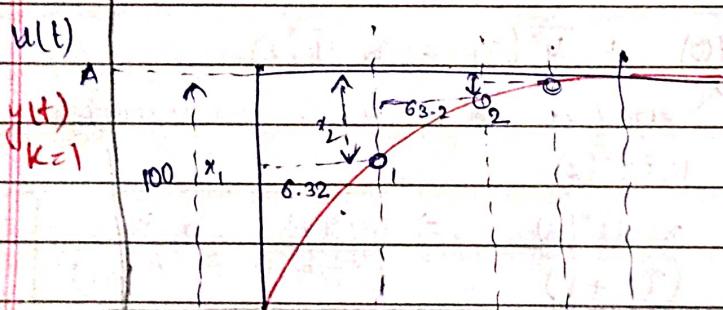
Point 1 is 63.2% of  $x_1$

Point 2 is 63.2% of

the remaining (that

is  $x_2$ ) of one  $\tau$

time constant before



$a=1$

In 4 time constants,  
we reach  $\approx 98.2\%$   
of overall remaining

$$t=1, y(t) = 0.632 y_{eq}$$

In 5 time constants,  $99.3\%$ .

Q: In what time will system reach new ss?

2E - 86.45%

$$\tau = a_1 / \omega_0 \text{ from system}$$

3E - 89.01%

$$4\tau \rightarrow 98.2\%$$

4\tau - 89.01%

$$5\tau \rightarrow 99.3\%$$

Infinite time  $\rightarrow 100\%$

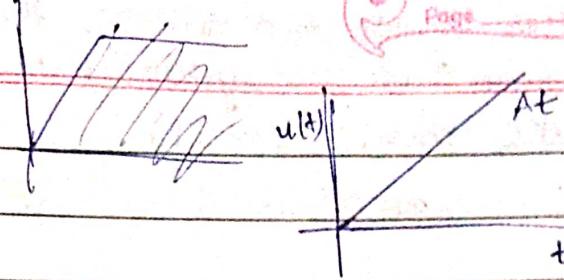
$\tau$ : time reqd to ~~reach~~ reach 63.2% of <sup>step</sup> forced input  $f^*$

Does not mean  $\tau$  is specifically linked to step f's.

Take another forcing f<sup>n</sup>

b) Ideal ramp function:

$$u(t) = \begin{cases} 0, & t < 0 \\ At, & t \geq 0 \end{cases}$$



$$g(x) = x$$

$$f(x) = u(e^{-\alpha x} + \beta/\alpha - 1)$$

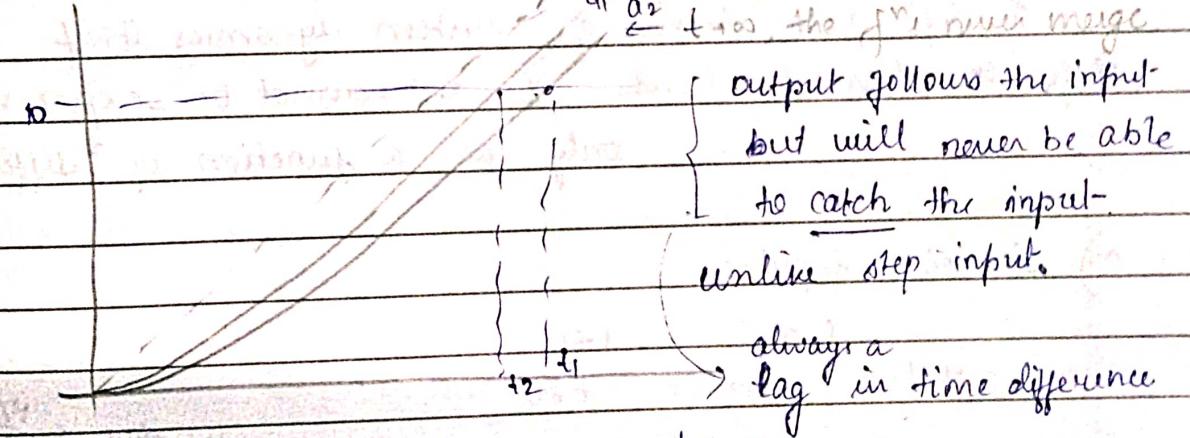
$$\tilde{Y}^*(s) = \frac{A}{(s+1)} \tilde{U}^*(s); \quad \tilde{U}^*(s) = \frac{A}{s^2}$$

$$\tilde{Y}^*(s) = \frac{KA}{2s+1} \left[ \frac{1}{s+1}, \frac{1}{s^2} \right]$$

$$= AK \left[ \frac{1}{2s+1} - \frac{1}{s+1} + \frac{1}{s^2} \right]$$

$$Ts^2 - Ts - \frac{A}{2} + \frac{A}{s+1}$$

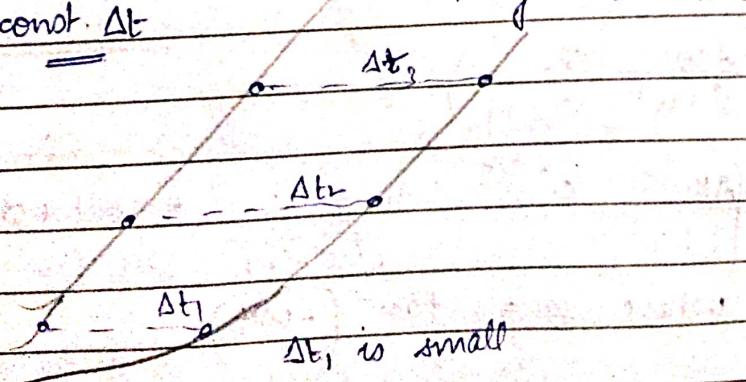
$$y(t) = AKT \left( e^{-t/\tau} + \frac{t}{\tau} - 1 \right)$$



We wanted "10". Reaches "10" after some time.

Goal: To reach const.  $\Delta t$

large  $\tau$   $\Rightarrow$  more lag



$\Delta t \uparrow$  then becomes constant

When will the  $\Delta t$  be constant? — after 52

Date \_\_\_\_\_  
Page \_\_\_\_\_

If system adjusts to a new SS, there will be transience in the system, but after some time, no transience occurs, then. ( $\sim 5\%$ )

so, concept of  $\tau$  is there for in other systems also.

• Time invariant :  $\frac{dy}{dt} = 0$  &  $\frac{du}{dt} = 0$

• Time independent : not a fn of "t"

$\Rightarrow$  In case of "ramp f", neither of the input or output are zero.

$\Rightarrow$  Here, the difference b/w  $u$  &  $y$  in slopes is invariant with time.

$$\Rightarrow \frac{d(u-y)}{dt} = 0$$

$$\frac{dy}{dt} \neq 0$$

$$\frac{du}{dt} \neq 0$$

System in absolute sense is never time invariant.

$\Rightarrow$  Step step input -

$\tau$  defined in terms of system dynamics itself.

$\Rightarrow$  for other functions,  $\tau$  cannot be expressed so, only as a function of  $t$  difference.

c) Sinusoidal input :

$$u(t) = \begin{cases} 0 & t < 0 \\ A \sin \omega t & t \geq 0 \end{cases}$$

$$\bar{U}(s) = \frac{A\omega}{s^2 + \omega^2}$$

$$\bar{Y}(s) = \frac{A\omega}{s^2 + \omega^2} \left( \frac{\omega}{s + j\omega} \right)$$

$$y(t) = A \left[ \frac{A\omega}{1 + \tau^2 \omega^2} e^{-t/\tau} + \frac{1}{\sqrt{\omega^2 \tau^2 + 1}} \sin(\omega t + \phi) \right]$$

$$\text{where } \phi = \tan^{-1}(-\omega \tau)$$

transient cover-up

$$A\tau = \lim_{s \rightarrow \infty} (s - \tau) Y(s)$$

$$\phi = 0 \Rightarrow \theta = 0 \quad (\text{no lag})$$

Ultimate response:

large  $t$

monosoidal

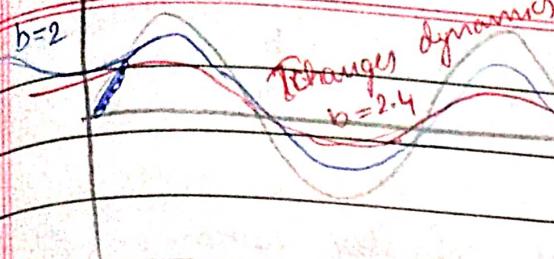
const. difference

diff.  $\omega$ , amplitude

Date \_\_\_\_\_  
Page \_\_\_\_\_

output,  $\phi = 0$

freq



Transient dynamics

$b=2.4$

$\phi=0$

Step

Ramp

tries to catch  
but delay

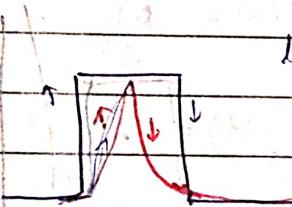
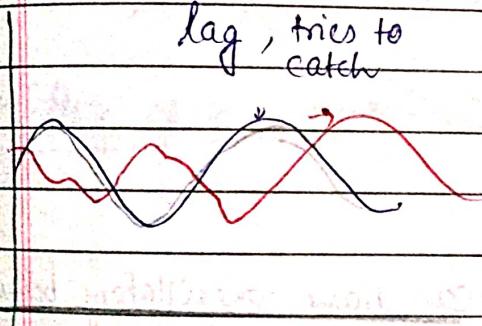
trying to catch  
forever

monosoid

pulse

lag, tries to  
catch

lag, tries to catch



Impulse

This happens because of non-zero  
time constant,  $\tau$ .

lag

If  $\tau \rightarrow 0$ , system responds very quickly  
 $\Rightarrow$  system will adapt to new state  
w/o transience.

$$\frac{d}{dt}y + \gamma = Ku(t)$$

$$g(s) = \frac{\gamma}{s + \gamma}$$

Pure gain systems.

when,  $\gamma = 0$ ,  $g(s) = K \rightarrow$  only gain  
in system

& no time const.,

$$a_1 \frac{dy}{dt} + a_0 y = bu(t)$$

Consider  $a_0 = 0$

$$\frac{dy}{dt} = b u(t) = K_1 u(t) ; K_1 = \frac{b}{a_1}$$

$$sY(s) = K_1 U(s)$$

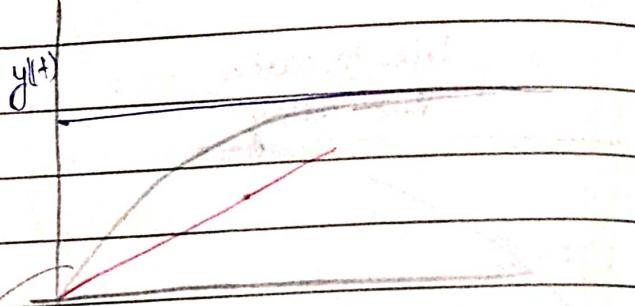
$$g(s) = \frac{K_1}{s} \quad - \text{pure capacitor system}$$

a) Step input :  $U(s) = A$

$$\bar{Y}(s) = AK$$

$$Y(t) = AKt$$

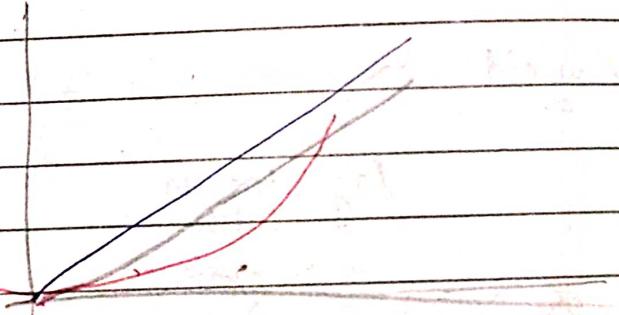
$$AK(1 - e^{-\tau t})$$



b) Ramp :  $\bar{Y}(s) = \frac{A}{s^2}$

$$\bar{Y}(s) = \frac{AK}{s^3}$$

$$\Rightarrow y(t) = AK t^2$$



## SECOND ORDER SYSTEM

- can have oscillatory behavior

$$a_2 \frac{d^2y}{dt^2} + a_1 \frac{dy}{dt} + a_0 y = bu(t)$$

$$\left(\frac{a_2}{a_0}\right) \frac{d^2y}{dt^2} + \left(\frac{a_1}{a_0}\right) \frac{dy}{dt} + y = \left(\frac{b}{a_0}\right) u(t)$$

$$\frac{a_2}{a_0} = \xi^2, \quad \xi = \text{natural period of oscillation}$$

$\frac{a_1}{a_0} = 2\xi\zeta$  ;  $\zeta$  = damping coefficient. - signifies whether the response to new change will be oscillatory  
etc.

$$\frac{b}{a_0} = K ; K = \text{gain}$$

$$\frac{d^2y}{dt^2}(s)$$

Deviat & LT

$$s^2(y(s) - y(0)) - s^2y'(0) + 2\zeta s y(s) - y'(0) = K u(s)$$

$$g(s) = \frac{K}{s^2 + 2\zeta s + 1}$$

$$s^2 + 2\zeta s + 1 \leftarrow \text{soln of quadratic eqn}$$

↓  
partial fraction

asymptotic response, ↓ oscillatory

from state space analysis

2nd order systems show osc. response ; but they can also show that also b/w 2 SSs.

Problems → convert to standard form (forcing  $f^n$ : hT<sub>in</sub>)

### 1/3 Analysis of the transfer function:

• State-space domain:  $\frac{dx_1}{dt} = f_1(x_1, \dots)$

$$\frac{dx_2}{dt} = f_2(x_1, \dots)$$

$$\frac{dx_n}{dt} = f_n(x_1, \dots)$$

$$\frac{dx}{dt} = Ax \quad \leftarrow \text{Matrix Eqn}$$

↑ dynamical vector

Coeff matrix

null space → eqn solutions

eigen values → stability analysis

• Transform domain analysis:  $\frac{dy}{dt} = f(y, u) \quad \begin{matrix} \uparrow \\ \text{input variable} \end{matrix}$

$\downarrow$   
output variable

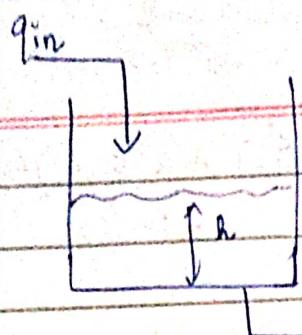
Laplace transform ( $g(s) \leftarrow \text{transfer f''}$ )

$$\Rightarrow g(s) = \frac{y(s)}{u(s)}$$

$$\Rightarrow y(t) = \mathcal{L}^{-1}[g(s)u(s)]$$

tells about  
dynamical features  
of the system

Order = 1



$$\Delta \frac{dh}{dt} = q_{in} - q_{out}$$

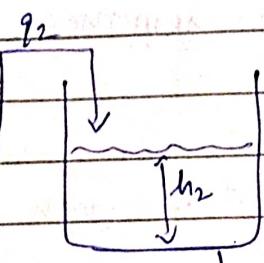
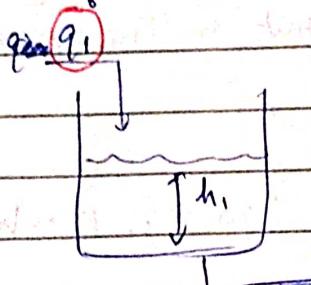
$$\frac{dh}{dt} = \frac{q_{in}}{\Delta} - \frac{q_{out}}{\Delta}$$

If we want to analyze dynamics of  $h$   
 $h$ : dynamical variable  $\rightarrow$  output variable

How will  $h$  change?

$\rightarrow$  either by changing  $q_{in}$  or  $q_{out}$

Now, if we have 2 tanks



$$A_1 \frac{dh_1}{dt} = q_1 - q_2 \quad \textcircled{1}$$

$$A_2 \frac{dh_2}{dt} = q_2 - q_3 \quad \textcircled{2}$$

Order = 2

Our interest can be

• Overall input & overall output of the system -

say, output variable =  $q_3$  (it might go to a column)

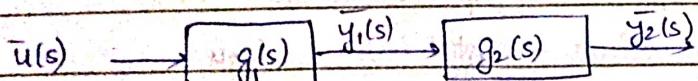
input variable =  $q_1$

Eqn ① is a 1<sup>st</sup> order ODE

$$g_1(s) = \frac{k_1}{\tau_1 s + 1}$$

Eqn ② is a 1<sup>st</sup> order ODE

$$g_2(s) = \frac{k_2}{\tau_2 s + 1}$$



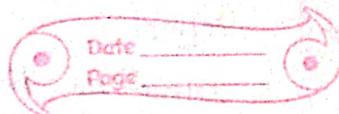
$$g_1(s) = \frac{\bar{y}_1(s)}{\bar{u}(s)}$$

$$g_2(s) = \frac{\bar{y}_2(s)}{\bar{y}_1(s)}$$

$$\Rightarrow g_1(s) g_2(s) = \frac{\bar{y}_2(s)}{\bar{u}(s)} \leftarrow \begin{matrix} \text{final output} \\ \bar{u}(s) \leftarrow \text{first input} \end{matrix}$$

Overall transfer  $f^n$  for  $n^{th}$  order system:

$$\bar{g}(s) = \prod_{i=1}^n g_i(s)$$



For a two-tank system,  $g(s) = \frac{k_1 k_2}{(\tau_1 s + 1)(\tau_2 s + 1)}$  → 2<sup>nd</sup> order

→ Def<sup>n</sup> of order in State space domain : no. of 1st order ODEs

→ " " in Transform domain : degree of polynomial of the denominator of the transfer f<sup>n</sup>.

i) Two tanks in a series :

$$g(s) = \frac{k_1 k_2}{(\tau_1 s + 1)(\tau_2 s + 1)}$$

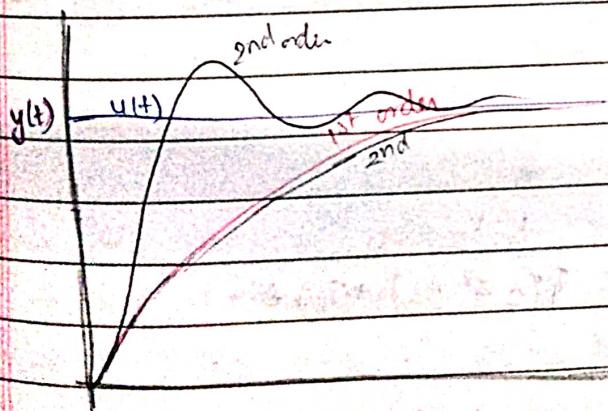
ii) Spring-mass system :

$$g(s) = \frac{k}{\tau^2 s^2 + 2\zeta\tau s + 1}$$

1<sup>st</sup> order  $g_1(s) = \frac{k}{\tau s + 1}$

2<sup>nd</sup> order :  $\frac{k}{\tau^2 s^2 + 2\zeta\tau s + 1}$

We want to know the difference b/w these two ↑  
system subjected to a unit step change.



2<sup>nd</sup> order: we have monotoneous  
oscillatory

We also saw this in S-S  
domain

↑ pure im → oscillatory

↑ complex → " (---)

↑ real → monotoneous

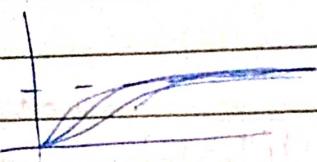
$$y(t) = \frac{\bar{y}(s)}{\bar{u}(s)} = \frac{K}{(\tau_1 s + 1)(\tau_2 s + 1)}$$

$$\bar{u}(s) = 1 \quad y(t) = L^{-1} \left\{ \frac{K}{s(\tau_1 s + 1)(\tau_2 s + 1)} \right\}$$

$$y(t) = K \left\{ 1 - \frac{\tau_2}{\tau_1 - \tau_2} e^{-t/\tau_1} - \left( \frac{\tau_2}{\tau_2 - \tau_1} \right) e^{-t/\tau_2} \right\}$$

Desmos:

whatever the values of  $T_1, T_2$  — no oscillations in system



### Qualitative analysis:

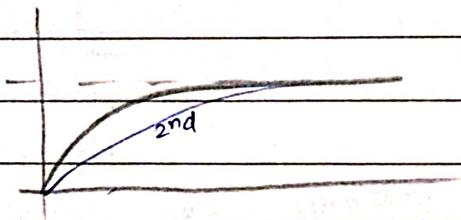
→ 2<sup>nd</sup> order transfer  $f^n$  obtained by multiplication of 1<sup>st</sup> order transfer f's never show oscillations.  
※ 1<sup>st</sup> order systems in series

→ How does it compare against 1<sup>st</sup> order dynamics.

$$g(s) = \frac{K}{2s+1}$$

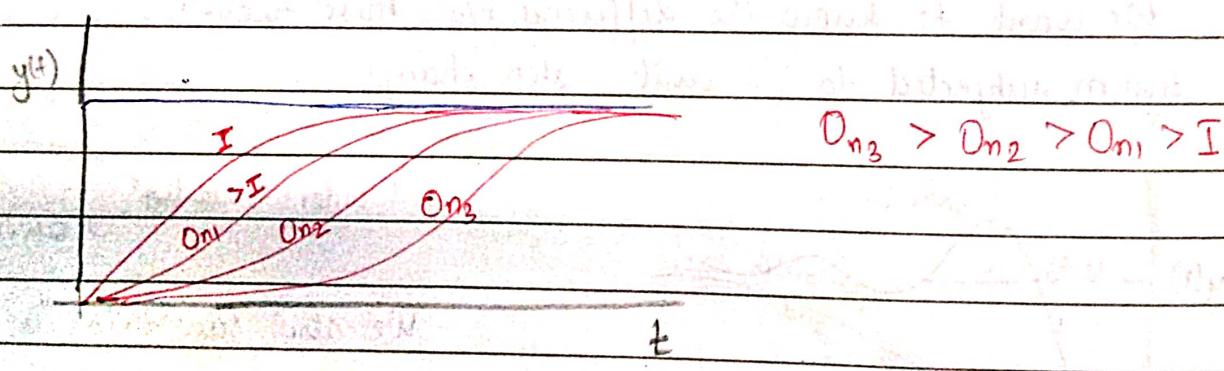
$$\bar{y}(s) = \frac{K}{2s+1}$$

$$\Rightarrow y(t) = K(1 - e^{-t/2})$$



Never does the 2<sup>nd</sup> order faster than 1<sup>st</sup> order.

A higher order system is always slower than 1<sup>st</sup> order sys.



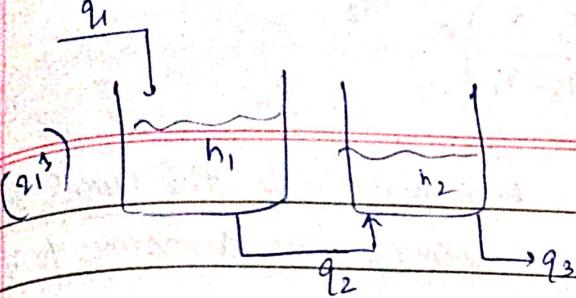
$$g(s) = \frac{K_1 K_2}{(\zeta_1 s + 1)(\zeta_2 s + 1)} = \frac{K_1 K_2}{\zeta_2 s^2 + (\zeta_1 + \zeta_2)s + 1}$$

↓ corresponds to

$$\bar{u}(s) \rightarrow \boxed{\bar{g}_1(s)} \xrightarrow{\bar{y}_1(s)} \boxed{\bar{g}_2(s)} \rightarrow \bar{y}_2(s)$$

This means that the level of liquid in tank 1 is not affected by the level of liquid in tank 2.

Non-interacting systems



There can be a backflow depending on the levels of  $h_1$  &  $h_2$

$$y_1 = f(y_1, y_2, u)$$

$$y_2 = f(y_1, y_2)$$

$$\frac{a_1}{dt} \frac{dy_1}{dt} + a_2 y_1 + a_3 y_2 = a_4 u$$

$$b_1 \frac{dy_1}{dt} + b_2 y_2 - b_3 y_1 = 0$$

$$\Rightarrow \frac{a_1}{a_2} \frac{dy_1}{dt} + \frac{a_3}{a_2} y_2 = \frac{a_4}{a_2} y_1 \Rightarrow \left( \frac{a_1}{a_2} s + 1 \right) \bar{y}_1(s) + \frac{a_3}{a_2} \bar{y}_2(s) = \frac{a_4}{a_2} \bar{y}_1(s)$$

$$\Rightarrow \frac{b_1}{b_2} \frac{dy_2}{dt} - \frac{b_3}{b_2} y_1 + y_2 = 0 \Rightarrow \left( \frac{b_1}{b_2} s + 1 \right) \bar{y}_2(s) - \frac{b_3}{b_2} \bar{y}_1(s) = 0$$

Simultaneously solve the algebraic eqn's for  $\bar{y}_1, \bar{y}_2$

~~dynamics of the 1st tank~~

$$\frac{\bar{y}_1(s)}{\bar{u}(s)} = \begin{bmatrix} a_4 b_1 s + b_2 a_4 \\ a_2 b_2 + a_3 b_3 \end{bmatrix}$$

~~output of 1st tank~~  $\frac{\text{input}}{(a_2 b_2 + a_3 b_3)} \left( \frac{a_1 b_1}{a_2 b_2 + a_3 b_3} \right) s^2 + \left( \frac{a_1 b_2 + a_2 b_1}{a_2 b_2 + a_3 b_3} \right) s + 1$

$$\frac{\bar{y}_2(s)}{\bar{u}(s)} = \frac{(a_4 b_3)}{(a_2 b_2 + a_3 b_3)}$$

~~overall dynamics~~  $\frac{(a_1 b_1)}{(a_2 b_2 + a_3 b_3)} s^2 + \left( \frac{a_1 b_2 + a_2 b_1}{a_2 b_2 + a_3 b_3} \right) s + 1$

Both are 2nd order dynamics

Not a multiplication of 2 TFs.

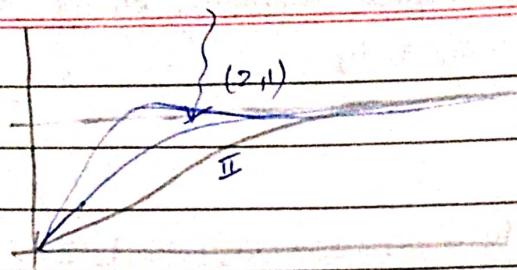
1st tank also 2nd order because it takes time to adjust to the disturbance from 2nd tank.

1st tank dynamics,  $g(s) = \frac{s^2 + 1}{(\zeta_1 s + 1)(\zeta_2 s + 1)}$

Order of the system  $\rightarrow (2,1)$

$$y(t) = L^{-1} \left\{ \frac{s^2 + 1}{s(\zeta_1 s + 1)(\zeta_2 s + 1)} \right\}$$

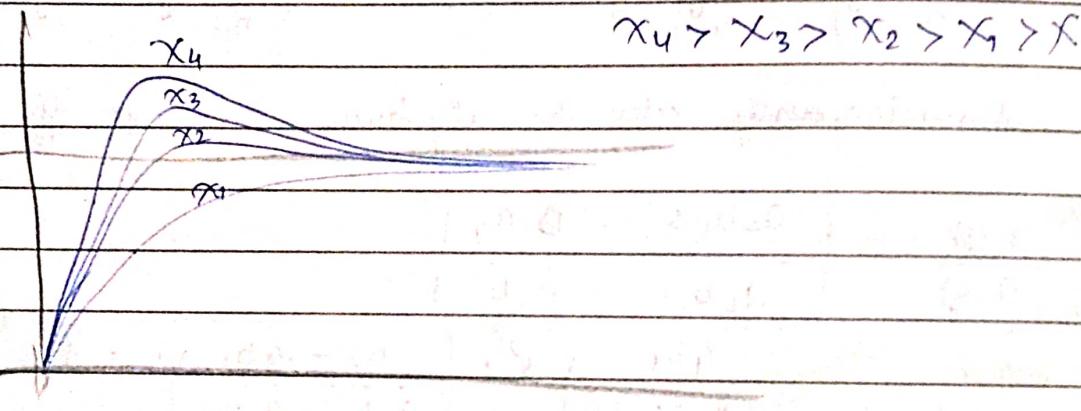
$$y(t) = 1 - \left( \frac{\tau_1 - x}{\tau_1 - \tau_2} \right) e^{-t/\tau_1} - \left( \frac{\tau_2 - x}{\tau_2 - \tau_1} \right) e^{-t/\tau_2}$$



Any quantity in the num' is pushing the dynamics to be faster.

$x$  is making system over-enthusiastic

~~and~~  $x$  is changing dynamics to an extent that  $\bar{u}(t)$  is also crossed. but ultimately comes to  $u(t)$ . Hence, it can overshoot.



$$g(s) = \frac{x_s + 1}{(z_p + 1)(z_n + 1)}$$

$\leftarrow$  zero       $\leftarrow$  poles

More poles - more sluggish  
more zeros - faster,  
enthusiastic

### Multiple inputs

$$\frac{dx_1}{dt} = a_{11}x_1 + a_{12}x_2 + \dots + a_{1n}x_n + b_{11}u_1 + b_{12}u_2 + \dots + b_{1m}u_m$$

$$\frac{dx_n}{dt} = a_{nn}x_1 + a_{2n}x_2 + \dots + a_{nn}x_n + b_{n1}u_1 + b_{n2}u_2 + \dots + b_{nm}u_m$$

$$\Rightarrow y_1 = c_{11}x_1 + c_{12}x_2 + \dots + c_{1n}x_n + d_{11}u_1 + d_{12}u_2 + \dots + d_{1m}u_m$$

no. of outputs

$$y_p = c_{p1}x_1 + c_{p2}x_2 + \dots + c_{pn}x_n + d_{p1}u_1 + d_{p2}u_2 + \dots + d_{pm}u_m$$

$m$ -dynamical variables  
 $m$ -input variables  
 $p$ -output variables

MIMO

Date \_\_\_\_\_  
Page \_\_\_\_\_

$$\frac{dx}{dt} = Ax + Bu$$

$$\frac{dy}{dt} = cx + du$$

Transform

$$s\bar{x}_1 = a_{11}\bar{x}_1 + a_{12}\bar{x}_2 + \dots + a_{1n}\bar{x}_n + b_{11}\bar{u}_1 + b_{12}\bar{u}_2 + \dots + b_{1m}\bar{u}_m$$

or

!

$$s\bar{x}_n = a_{n1}\bar{x}_1 + a_{n2}\bar{x}_2 + \dots + a_{nn}\bar{x}_n + b_{n1}\bar{u}_1 + b_{n2}\bar{u}_2 + \dots + b_{nm}\bar{u}_m$$

$$\bar{y}_1 = c_{11}\bar{x}_1 + c_{12}\bar{x}_2 + \dots + c_{1n}\bar{x}_n + d_{11}\bar{u}_1 + d_{12}\bar{u}_2 + \dots + d_{1m}\bar{u}_m$$

!

!

$$\bar{y}_p = c_{p1}\bar{x}_1 + c_{p2}\bar{x}_2 + \dots + c_{pn}\bar{x}_n + d_{p1}\bar{u}_1 + d_{p2}\bar{u}_2 + \dots + d_{pm}\bar{u}_m$$

$n+p$  equations to be solved simultaneously.

$$\left( \begin{array}{ccccc} 1 & 0 & 0 & \dots & 0 \\ 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ 0 & 0 & 0 & \dots & 1 \end{array} \right) \left( \begin{array}{c} a_{11} & a_{12} & \dots & a_{1n} \\ | & | & & | \\ a_m & a_m & \dots & a_m \end{array} \right) \left( \begin{array}{c} \bar{x}_1 \\ \bar{x}_2 \\ \vdots \\ \bar{x}_n \end{array} \right)$$

$$\left( \begin{array}{cccc} b_{11} & b_{12} & \dots & b_{1m} \\ | & | & & | \\ b_{m1} & b_{m2} & \dots & b_{mm} \end{array} \right) \left( \begin{array}{c} \bar{u}_1 \\ \bar{u}_2 \\ \vdots \\ \bar{u}_m \end{array} \right)$$

$$\Rightarrow (SI - A)\bar{x} = B\bar{u}$$

$$\bar{x} = (SI - A)^{-1} B \bar{u} \quad \leftarrow \text{Laplace transform of dynamical variable}$$

$$\begin{bmatrix} \bar{y}_1 \\ \bar{y}_2 \\ \vdots \\ \bar{y}_p \end{bmatrix} = \begin{bmatrix} C_{11} & C_{12} & \dots & C_{1n} \\ \vdots & \vdots & & \vdots \\ C_{p1} & C_{p2} & \dots & C_{pn} \end{bmatrix} \begin{bmatrix} \bar{x}_1 \\ \bar{x}_2 \\ \vdots \\ \bar{x}_n \end{bmatrix} + \begin{bmatrix} d_{11} & d_{12} & \dots & d_{1m} \\ \vdots & \vdots & & \vdots \\ d_{p1} & d_{p2} & \dots & d_{pm} \end{bmatrix} \begin{bmatrix} \bar{u}_1 \\ \bar{u}_2 \\ \vdots \\ \bar{u}_m \end{bmatrix}$$

$$\bar{y}(s) = \underline{\underline{C}} \bar{x} + \underline{\underline{D}} \bar{u}$$

$$\bar{y}(s) = \underline{\underline{C}} \{ \underline{\underline{S}} \underline{\underline{I}} - \underline{\underline{A}} \}^{-1} \underline{\underline{B}} \bar{u} + \underline{\underline{D}} \bar{u}$$

Transfer  $\frac{\bar{Y}(s)}{\bar{U}(s)} = \underline{\underline{C}} \{ \underline{\underline{S}} \underline{\underline{I}} - \underline{\underline{A}} \}^{-1} \underline{\underline{B}} + \underline{\underline{D}}$

For a SISO system  $\rightarrow$  transfer  $f^n$

For a MIMO system  $\rightarrow$  transfer  $f^n$  matrix

Interconversion of state space  $\leftrightarrow$  Transfer domain

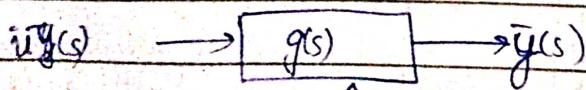
$$\left. \begin{array}{l} \frac{d\bar{x}}{dt} = \underline{\underline{a}}\bar{x} + \underline{\underline{b}}\bar{u} \\ \bar{y} = \underline{\underline{c}}\bar{x} + \underline{\underline{d}}\bar{u} \end{array} \right\} \text{state-space domain model}$$

$$s\bar{x}(s) = \underline{\underline{a}}\bar{x}(s) + \underline{\underline{b}}\bar{u}(s) \quad | \quad \bar{y}(s) = \underline{\underline{c}}\bar{x}(s) + \underline{\underline{d}}\bar{u}(s)$$

$$\Rightarrow \bar{x}(s) = \frac{\underline{\underline{b}}}{(s-\underline{\underline{a}})} \bar{u}(s) \quad | \quad \Rightarrow \bar{y}(s) = \frac{\underline{\underline{c}}\underline{\underline{b}}}{(s-\underline{\underline{a}})} \bar{u}(s) + \underline{\underline{d}}\bar{u}(s)$$

$$\Rightarrow \bar{y}(s) = \frac{\underline{\underline{c}}\underline{\underline{b}} + \underline{\underline{d}}}{s-\underline{\underline{a}}} \bar{u}(s)$$

$$\bar{g}(s) = \frac{\bar{y}(s)}{\bar{u}(s)} = \frac{d\underline{\underline{s}} - ad + cb}{s-a}$$



Get  $\bar{g}(s)$  from block diagram:

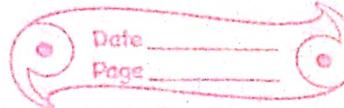
$$\frac{\bar{y}(s)}{\bar{u}(s)} = \frac{ds + (cb-ad)}{s-a}$$

$$s\bar{y}(s) - a\bar{y}(s) = ds \cancel{\bar{g}(s)} \bar{u}(s) + (cb-ad) \bar{u}(s)$$

$\frac{dy}{dt}$  must come from  $\frac{dy}{dt}$

$$\Rightarrow \frac{dy}{dt} - ay = \left( \frac{du}{dt} \right) d + (cb - ad)u$$

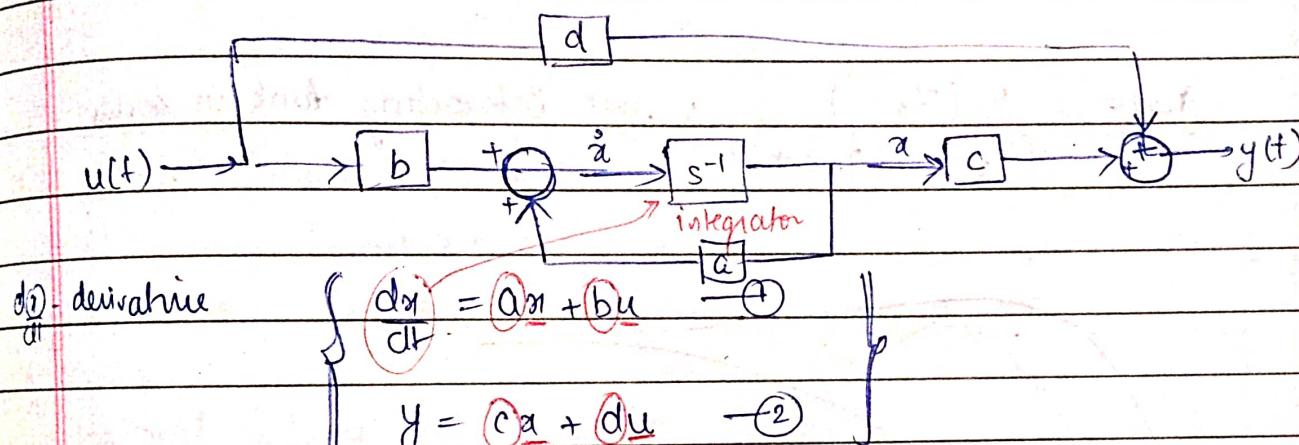
↓  
output      input      state-space domain model.



We have converted all outputs in terms of inputs & derivatives.

### Block diagram

- should have input, output, & those operations



Integrator :  $s^{-1}$

Relationship b/w  $a$  &  $\alpha$  is st. if we supply  $\frac{dx}{dt}$  to  $\frac{du}{dt}$  with

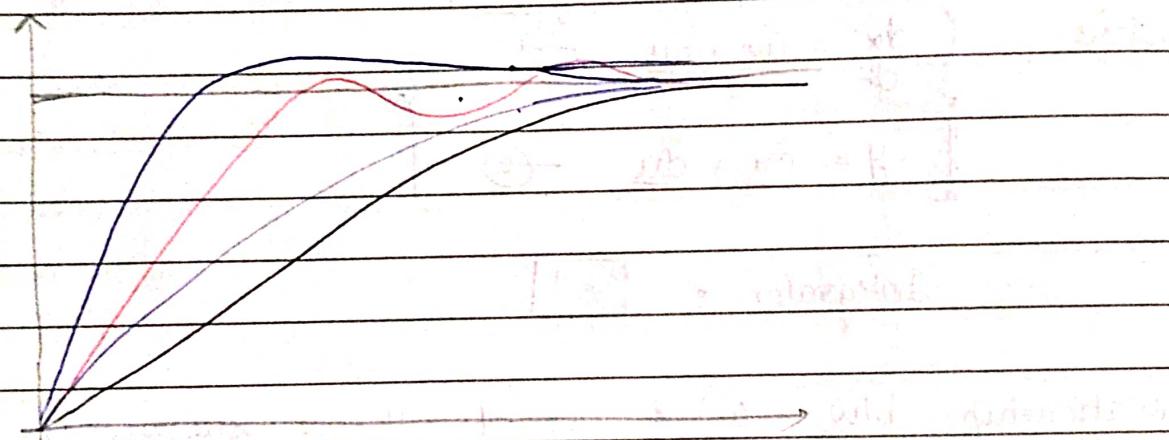
19/3/21 Analysis of Transfer Functions:

$$g(s) = \frac{K}{2s+1} \quad ; \text{ liquid level in tank}$$

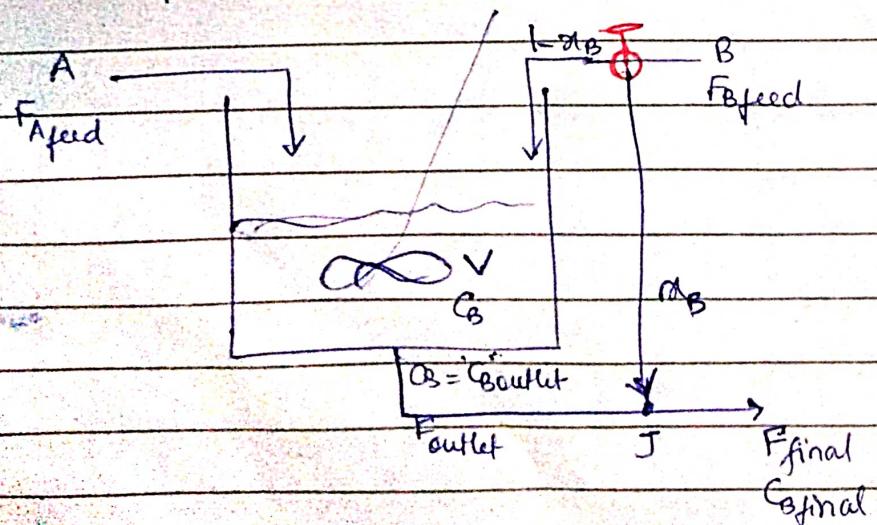
$$g(s) = \frac{K}{T^2 s^2 + 2\zeta T s + 1} \quad ; \text{ spring-mass}$$

$$g(s) = \frac{K}{(T_1 s + 1)(T_2 s + 1)} \quad ; \text{ two tanks (non-interacting) in series}$$

$$g(s) = \frac{K(x_1 + 1)}{(T_1 s + 1)(T_2 s + 1)} \quad ; \text{ two interacting tank in series}$$



- Q. Consider a mixing tank in which a small quantity of liquid B is added to a large volumetric flow rate of liquid A. A fraction  $a$  of B is sent to the outlet of tank.



Control valve to ~~acc~~ get desired  $C_{final}$

*Date \_\_\_\_\_  
Page \_\_\_\_\_*

Material balance around the tank:

$$\text{input} - \text{output} = \text{accumulation}$$

At steady state,

$$F_A\text{feed} + F_B\text{feed} = F_{\text{outlet}}$$

$$\text{Since, } F_A\text{feed} \gg F_B\text{feed} \Rightarrow F_{\text{outlet}} \approx F_A\text{feed}$$

Material balance at the junction J:

$$F_{\text{outlet}} + \alpha_B F_B\text{feed} = F_{\text{final}}$$

$$\Rightarrow F_A\text{feed} + \alpha_B F_B\text{feed} = F_{\text{final}}$$

$$\because F_A\text{feed} \gg F_B\text{feed}$$

$$\Rightarrow F_{\text{outlet}} \approx F_{\text{final}} \approx F_A\text{feed} \quad \text{--- (1)}$$

Material balance for B in the tank:

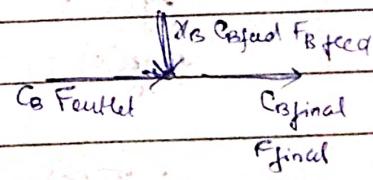
$$0 + (1-\alpha_B) F_B\text{feed} \cdot C_B\text{feed} - (F_{\text{outlet}}) C_B\text{outlet} = V \frac{dC_B}{dt}$$

$$(1-\alpha_B) F_B\text{feed} \cdot C_B\text{feed} - F_{\text{outlet}} C_B\text{outlet} = V \frac{dC_B}{dt}$$

$$\Rightarrow \frac{V}{F_A\text{feed}} \frac{dC_B}{dt} + C_B\text{outlet} = \left( \frac{F_B\text{feed} C_B\text{feed}}{F_A\text{feed}} \right) (1-\alpha_B) \quad \text{--- (2)}$$

At junction J:

$$C_B\text{outlet} + \alpha_B C_B\text{feed} F_A\text{feed} = C_{\text{final}} F_{\text{final}}$$



$$\Rightarrow C_B\text{outlet} + \alpha_B C_B\text{feed} F_A\text{feed} = C_{\text{final}} F_{\text{final}}$$

$$\Rightarrow C_B\text{outlet} + \alpha_B C_B\text{feed} F_A\text{feed} = C_{\text{final}} F_A\text{feed}$$

$$\Rightarrow C_{\text{final}} = C_B + \frac{F_A\text{feed} \alpha_B C_B\text{feed}}{F_A\text{feed}} \quad \text{--- (3)}$$

## Dynamical eq's:

$C_B \leftarrow$  dynamical variable =  $x$

$C_{Final} \leftarrow$  output variable =  $y$

$$\left\{ \begin{array}{l} \checkmark \frac{dC_B}{dt} + C_B = \left( \frac{\text{Fixed}}{\text{Fixed}} \frac{\text{Capacitance}}{\text{Capacitance}} \right) (1-x_B) \leftarrow \frac{dx}{dt} + C_0 x = bu \\ \text{Fixed } C_{\text{fixed}} \end{array} \right.$$

Fixed  $C_{\text{fixed}} \leftarrow$  input variable =  $u$

Fixed

$$\left\{ \begin{array}{l} \checkmark = T, K_B = (1-x_B), C_0 = 1 \\ \text{Fixed} \end{array} \right.$$

$$\Rightarrow T \frac{dx}{dt} + x = (1-x_B) u \quad \text{--- I}$$

$$\Rightarrow y = x + x_B u \quad \text{--- II}$$

Our domain cast in such a form that we can use  $\mathcal{Z}$ -transform domain analysis

$$\checkmark \bar{x}(s) + \bar{x}(s) = (1-x_B) \bar{u}(s)$$

$$\Rightarrow \bar{x}(s) = \frac{(1-x_B) \bar{u}(s)}{2s+1}$$

$$\bar{y}(s) = \bar{x}(s) + x_B \bar{u}(s)$$

$$\Rightarrow \bar{y}(s) = \left[ \frac{1-x_B}{2s+1} + x_B \right] \bar{u}(s)$$

$$\Rightarrow \frac{\bar{y}(s)}{\bar{u}(s)} = \frac{1-x_B + 2x_B s + x_B}{2s+1}$$

$$\Rightarrow \frac{\bar{y}(s)}{\bar{u}(s)} = \frac{(x_B 2) s + 1}{(2s+1)} = \frac{x_B s + 1}{(2s+1)}$$

The transfer fn of the system has the form :

$$(1,1) \text{ order system} \quad g(s) = \frac{x_B s + 1}{2s+1}$$

$$g(s) = \frac{1}{\tau_s + 1} + \frac{x_s}{\tau_s + 1}$$

This system is similar to having 2 systems in series

dep input :  $u(s) = \frac{A}{s} = \frac{1}{s}$

$$\textcircled{1} \quad y(s) = \frac{1}{s} \left( \frac{x_s + 1}{\tau_s + 1} \right)$$

$$\Rightarrow y(t) = t^{-1} \left[ \dots \right]$$

$$y(t) = 1 - (1 - x_s) e^{-\alpha/\tau_s}$$

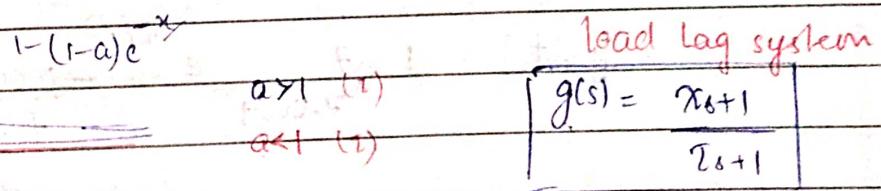
lead time const

lag time const

Des

Pure capacitive response - quickly follows input

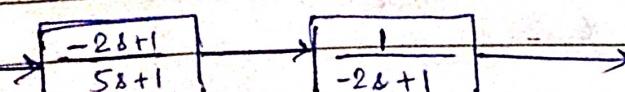
then, pure capacitive response superimposed on 1st order dynamics



$$\text{eg: } g(s) = \frac{1}{5s + 1}$$

We need to look at what & how the system is

composed of -



Lead-lag system

$g_1(s)$

$g_2(s)$

Pole-zero cancellation

$$g(s) = g_1(s) g_2(s) = \frac{-2s+1}{5s+1} \times \frac{1}{-2s+1} = \frac{1}{5s+1}$$

Poles of 1 component canceled zeros of 2<sup>nd</sup> component  
We could've imagined a 1st order dynamics  
from  $g(s)$ .  
But it is not so!



( $-2s+1$ ) → Under what conditions / when do we have poles & zeroes in RHP and LHP?  
→ Feedback comes from different direction?

Take  $g_2(s) = \frac{1}{-2.0001s + 1}$

Overall TF:  $g(s) = \frac{-2s+1}{5s+1} \times \frac{1}{-2.0001s+1}$

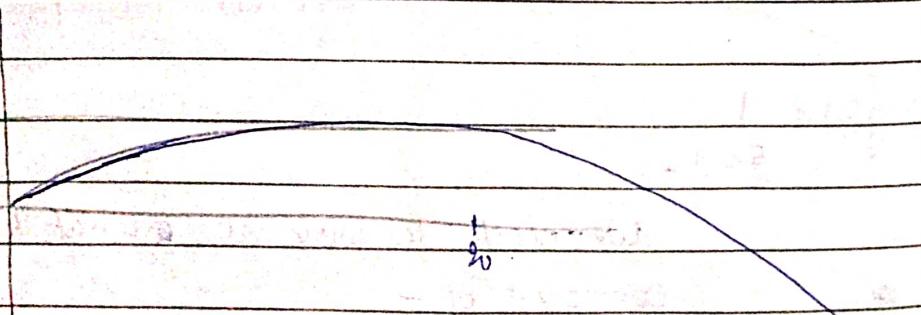
$$g_3(s) = \frac{-2s+1}{-10.0005s^2 + 2.9999s + 1}$$

$$g_4(s) = \frac{1}{5s+1}$$

Response to unit step function:

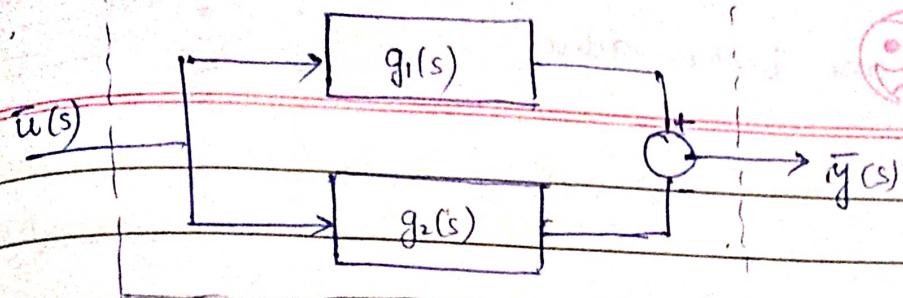
$$y_4(t) = 1 - e^{-t/5}$$

$$y_3(t) = 1 - \frac{7}{7.0001} e^{-t/5} - \frac{0.0001}{7.0001} e^{t/2.0001}$$



- \* We should also worry about how the TF has come

Systems were arranged in series "multiplication" of TFs



effect of  $g_1$  is to amplify the signal  
 $g_2$  is to dampen " "  
 (opposes)

$$g(s) = g_1(s) - g_2(s)$$

$$= \frac{K_1}{\tau_1 s + 1} - \frac{K_2}{\tau_2 s + 1}$$

$\uparrow$  main mode       $\uparrow$  opposition mode

$$\bar{y}(s) = \bar{u}(s) g_1(s) - \bar{u}(s) g_2(s)$$

$$\Rightarrow \bar{y}(s) = \frac{K_1}{s(\tau_1 s + 1)} - \frac{K_2}{s(\tau_2 s + 1)}$$

$$y(t) = L^{-1} \left[ \frac{K_1}{s(\tau_1 s + 1)} \right] - L^{-1} \left[ \frac{K_2}{s(\tau_2 s + 1)} \right]$$

$$y(t) = A K_1 (1 - e^{-t/\tau_1}) - A K_2 (1 - e^{-t/\tau_2})$$

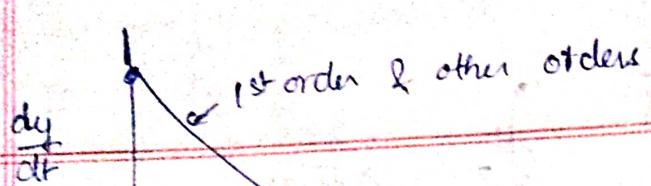
If the transfer function is of the above form, then what would be the ultimate response of the system.

$$\lim_{t \rightarrow \infty} y(t) = A(K_1 - K_2)$$

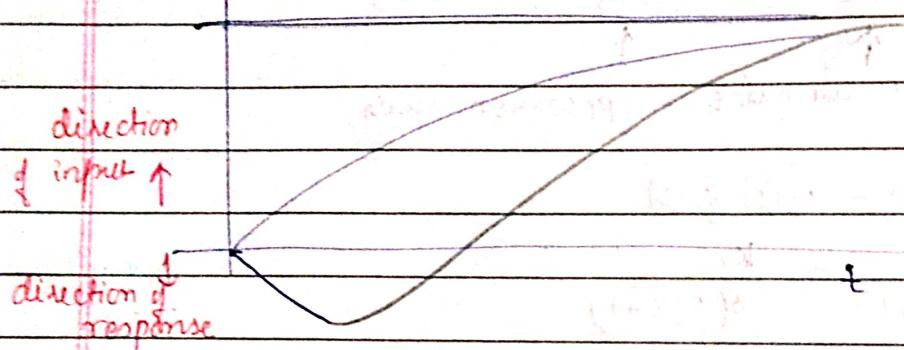
$$\frac{dy}{dt} = \frac{A K_1}{\tau_1} e^{-t/\tau_1} - \frac{A K_2}{\tau_2} e^{-t/\tau_2}$$

$$\left. \frac{dy}{dt} \right|_{t=0} = A \frac{K_1}{\tau_1} - \frac{K_2}{\tau_2}$$

$K_1 > K_2 \leftarrow$  known  
 But what about  $\tau_1$  &  $\tau_2$ ?  $\leftarrow$  no comment on this

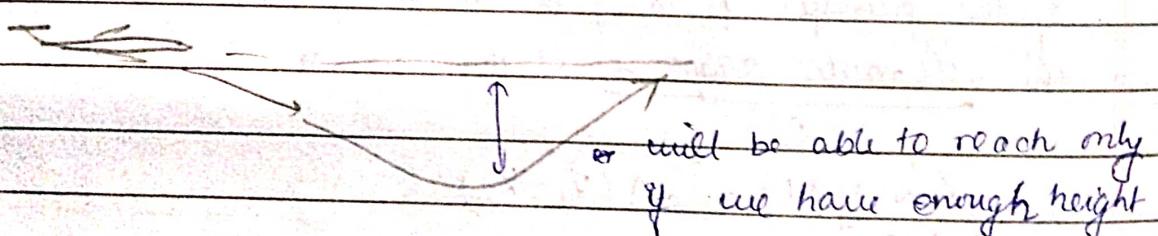


$$\text{Here, } \frac{dy}{dt} \begin{cases} > 0 & \text{if } \frac{k_1}{\tau_1} > \frac{k_2}{\tau_2} \\ < 0 & \text{if } \frac{k_1}{\tau_1} < \frac{k_2}{\tau_2} \end{cases}$$



The fate of the system as  $t \rightarrow \infty$  will always be  $\Rightarrow$  known before.

But initially, there is a negative slope. Because it has to reach to a +ve value, there will be inversion.



flattens because of  $\frac{k_1}{\tau_1} < \frac{k_2}{\tau_2}$

- If opposition is more, effect of opposition is more.
- A larger magnitude will cause larger delay in response.
- In such systems, we may or may not have inverse response.

$$g(s) = \frac{k_1}{\tau_1 s + 1} - \frac{k_2}{\tau_2 s + 1}$$

$$= \frac{k_1(\tau_2 s + 1) - k_2(\tau_1 s + 1)}{(\tau_1 s + 1)(\tau_2 s + 1)}$$

$$= \frac{(k_1 \tau_2 - k_2 \tau_1)s + (k_1 - k_2)}{(\tau_1 s + 1)(\tau_2 s + 1)}$$

$$= \frac{k(s+1)}{(\tau_1 s + 1)(\tau_2 s + 1)}$$

$$X = K_1 \tau_2 - K_2 \tau_1$$

$$X = \frac{k_1 \tau_2 - k_2 \tau_1}{\tau_1 - \tau_2}$$

$$K = K_1 - K_2$$

Depending on  $K_1, K_2, \tau_1, \tau_2$ ,  $X$  can be positive or  $X$  can be negative.

$$n = 10. \frac{K_2 \tau_1 - K_1 \tau_2}{K_1 - K_2}$$

$$g(s) = K \left( \frac{-\eta s + 1}{(\tau_1 s + 1)(\tau_2 s + 1)} \right)$$

$$y(t) = L^{-1} \left[ \frac{K(-\eta s + 1)}{s(\tau_1 s + 1)(\tau_2 s + 1)} \right]$$

$$y(t) = AK \left[ 1 + \frac{(\eta + \tau_1)}{(\tau_2 - \tau_1)} e^{-t/\tau_1} + \frac{(\eta + \tau_2)}{(\tau_1 - \tau_2)} e^{-t/\tau_2} \right]$$

Ultimate response,  $t \rightarrow \infty$

$$\lim_{t \rightarrow \infty} y(t) = AK \cancel{\downarrow}$$

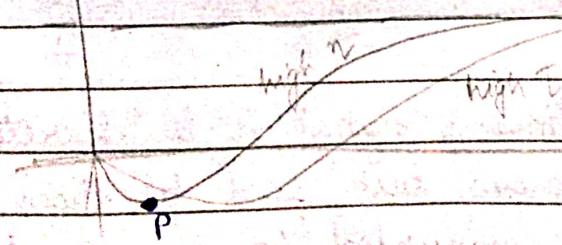
$\eta$  corresponds to zero  
 $\tau_1, \tau_2$  to poles

zeros

→ poles make dynamics sluggish

→ zeros make

dynamics faster



$$\tau_2 = 2.4$$

$$\tau_1 = 8.5$$

$$t = \frac{\tau_1 \tau_2 \ln(K_2 \tau_1)}{\tau_1 - \tau_2}$$

Slope inverts at P,  $\frac{dy}{dt} = 0$

$$\tau_2 = 1.1$$

$$K_1 = 4.9$$

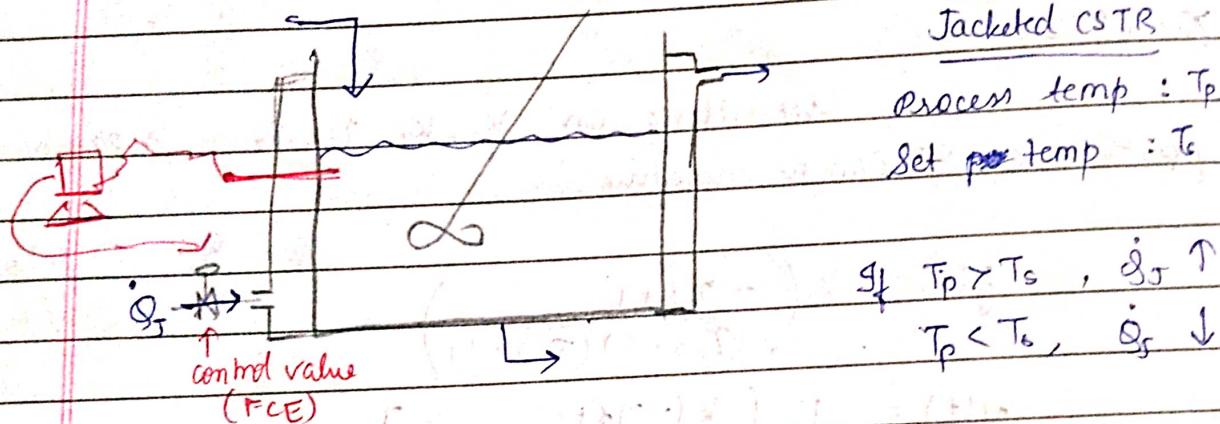
• Pure capacity / pure - depending on magnitudes.

26/3/21

Dynamics of discrete-time systemsDate \_\_\_\_\_  
Page \_\_\_\_\_

$$\bar{u}(s) \xrightarrow{\text{sg}(s)} \bar{y}(s) \rightarrow \mathcal{L}^{-1}\{y(s)\} = y(t) \rightarrow \text{dynamics of output variable}$$

$u(t)$ ,  $\alpha(t)$ ,  $y(t)$  → continuous functions (in time)



→ flow  $Q_s$  will change with  $T_p$  or  $(T_s - T_p)$  will come from model.

→ Model eq's: will be continuous ODEs.

• → depending on  $|T_s - T_p|$ , value position will change

$$\epsilon = T_p - T_s$$

→ How will you know  $\epsilon$ ?

We should have a temp measuring device, say thermocouple

→ The only  $\epsilon$  has to be sent to a controller - computer.

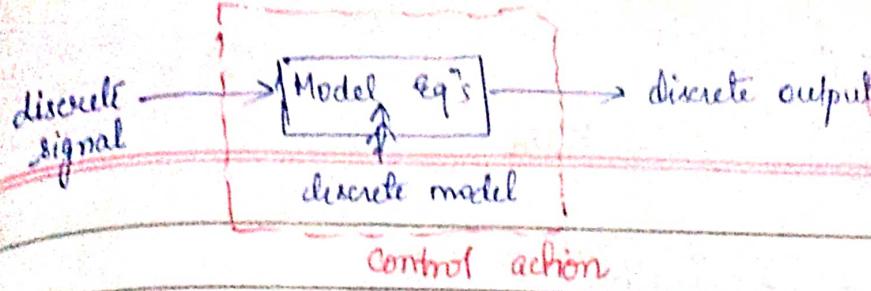
→ The only input to the controller has to be a digital signal for it to be processible. So, it has to be discrete.

→  $T_p$ : continuous signal

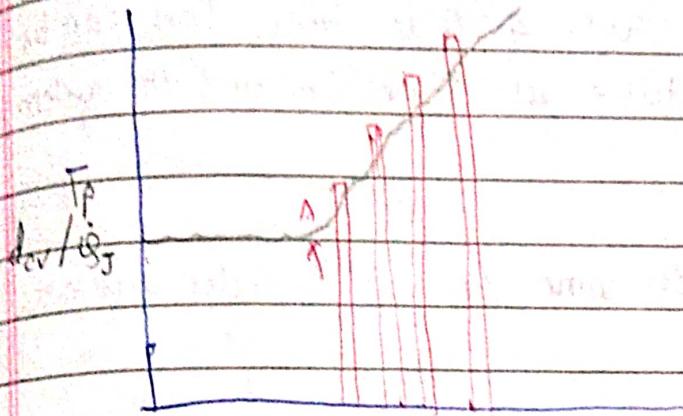
\* Conversion of a continuous signal to a discrete time signal.

\* Model eq's are continuous but feed to them are discrete so, they need to be converted to discrete time domain

\* Conversion of discrete time output signal to a continuous signal.



Date \_\_\_\_\_  
Page \_\_\_\_\_

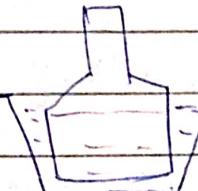


A - control action invoked

From A, FCE acts and diff values openings as  $T_p$  changes.

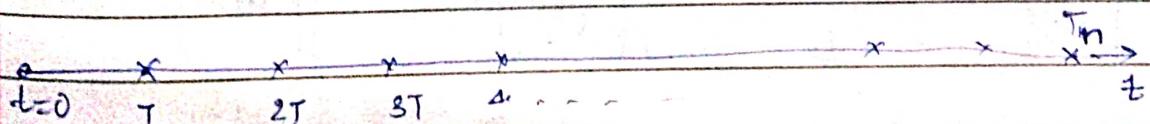
value open value close

\* We would like to have  
automatic control



- Value receives increasing  $T_p$  continuously as disturbance and is acted upon to regulate.
- But as model is discrete, output is discrete, so act" is discrete
- But we do not want this, so we should convert ~~it~~<sup>output</sup> to continuous domain.

### Conversion of continuous models to discrete-time models



$$\frac{dy}{dt} = \frac{y_{n+1} - y_n}{\Delta t}$$

$\Delta t = \Delta T$

$y_n$  : amplitude/ value of cont. f<sup>n</sup> at  $T_n$

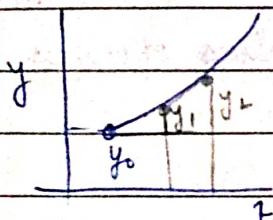
$$\frac{dy}{dt} = f(y, a)$$

$$\tau \frac{dy}{dt} + y = Ku(t)$$

$$\frac{T(y_{n+1} - y_n)}{\tau} + y_n = Ku_n$$

$$\Rightarrow y_{n+1} = \left(1 - \frac{T}{\tau}\right)y_n + \frac{KT u_n}{\tau}$$

This means that the amplitude ~~of~~ at time  $T_{n+1}$  can be obtained from the amplitude at time  $T_n$  and the system parameters.



Do same for 2nd order system.

Multiple input - Multiple output system:

$$\frac{dy_1}{dt} + a_{11}y_1 + a_{12}y_2 = b_{11}u_1 + b_{12}u_2$$

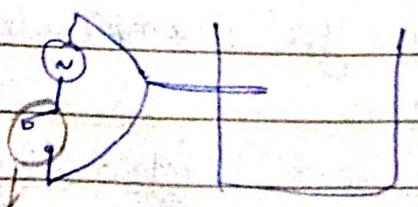
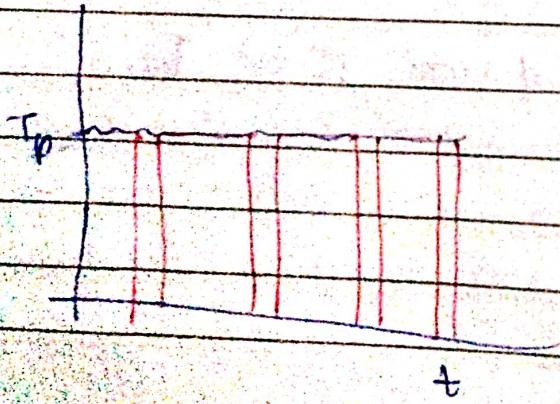
$$\frac{dy_2}{dt} + a_{21}y_1 + a_{22}y_2 = b_{21}u_1 + b_{22}u_2$$

$$\frac{y_{1,n+1} - y_{1,n}}{\tau} + a_{11}y_{1,n} + a_{12}y_{2,n} = b_{11}u_{1,n} + b_{12}u_{2,n}$$

$$\left\{ \frac{y_{2,n+1} - y_{2,n}}{\tau} + a_{21}y_{1,n} + a_{22}y_{2,n} = b_{21}u_{1,n} + b_{22}u_{2,n} \right.$$

$$\begin{bmatrix} y_{1,n+1} \\ y_{2,n+1} \end{bmatrix} = \begin{bmatrix} & & [y_{1,n}] \\ & & [y_{2,n}] \end{bmatrix} + \begin{bmatrix} & & [u_{1,n}] \\ & & [u_{2,n}] \end{bmatrix}$$

Conversion of a continuous input signal to a discrete time signal



input to it is continuous but we decide how to take output

## Limitation:

Date \_\_\_\_\_  
Page \_\_\_\_\_

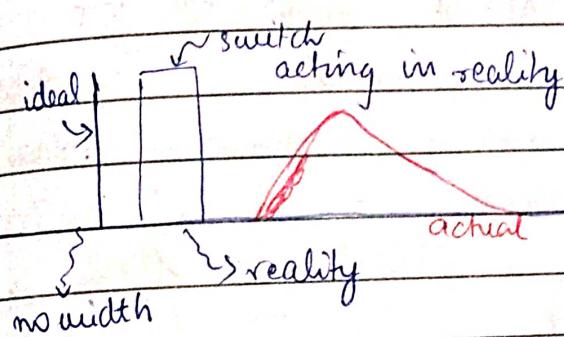
The observation here is const  $T_p$  but this may not be the reality.

- ✓ Sampling method
- ✓ System dependent on fineness of discretization.
- ✓ Correct sampling to retain resolution.

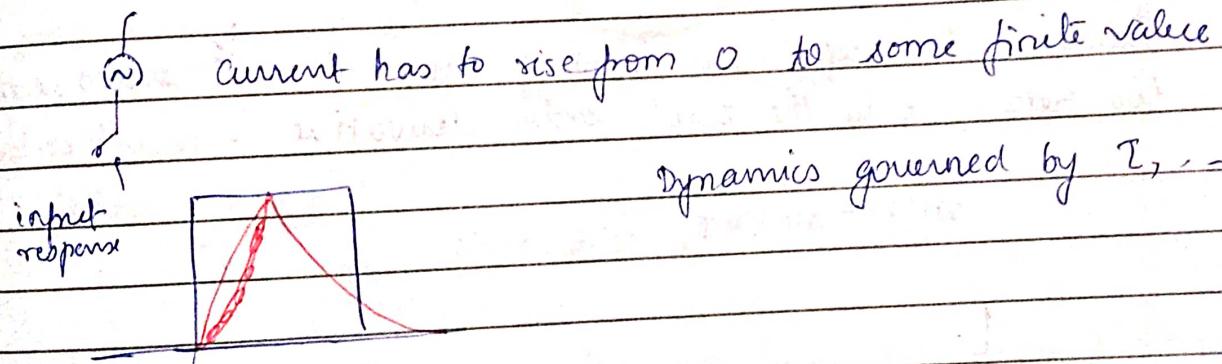
Oscillations captured



Value opened, stays for some time before closing.



But the actu of switch is like a pulse response.



discrete time

$$\text{discrete } y^*(t) = y^*(0) + y^*(T) + y^*(2T) + \dots$$

$$\therefore y^*(t) = y(0) + \delta(tT) + y(T)\delta(t-T) + \dots$$

cont.

$$y^*(t) = \sum_{n=0}^{\infty} y(nT) \delta(t-nT) \leftarrow \begin{array}{l} \text{discrete time signal} \\ \text{a number} \end{array}$$

$$\bar{y}^*(s) = \sum_{n=0}^{\infty} y(nT) L\delta(t-nT)$$

$$\Rightarrow \bar{y}^*(s) = \sum_{n=0}^{\infty} y(nT) e^{-nTs}$$

Conversion of a discrete output signal to a continuous input signal

$$m(\tau), m(2\tau), m(3\tau) \dots$$



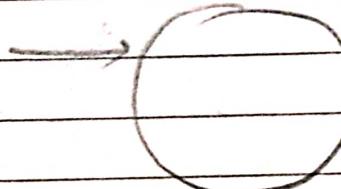
$$\text{continuous } \approx m(t)$$

Taylor expansion about  $nT$

$$m(t) = m(nT) + \frac{dm}{dt} \Big|_{nT} (t-nT) + \frac{1}{2} \frac{d^2m}{dt^2} \Big|_{nT} (t-nT)^2$$

about I know this discrete discrete discrete

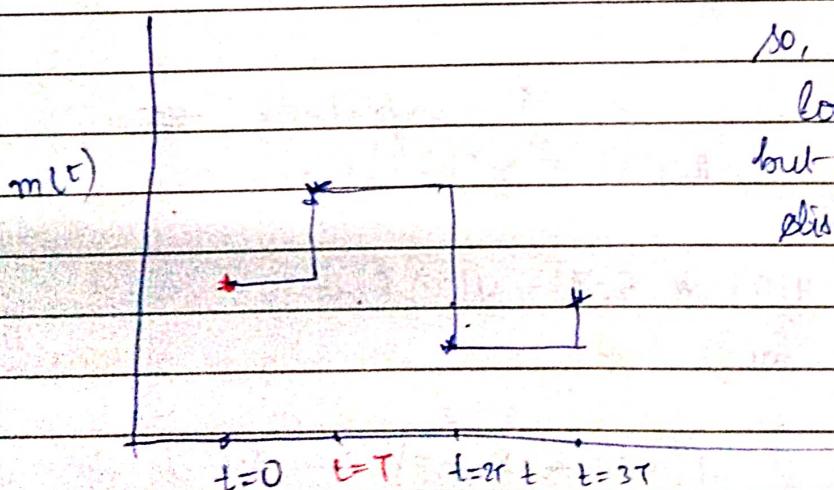
input  
discrete



continuous time

Problem: be careful with no. of terms in Taylor series expansion  
We retain only the zeroth order derivative - zeroth order hold

$$m(t) \approx m(nT)$$



so, our output  $f^n$  will look like a step function but it looks like a continuous discrete system, so not very useful.

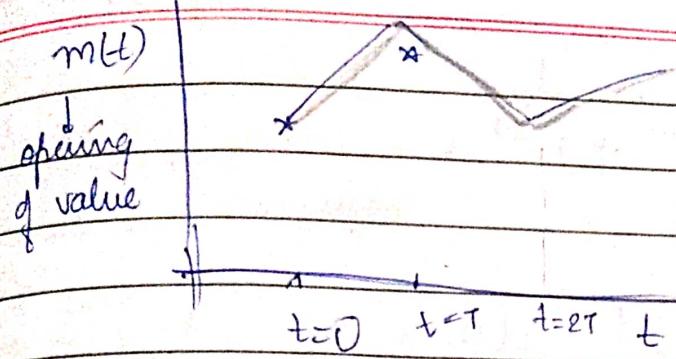
$$\text{1st order hold: } m(t) = m(nT) + \frac{dm}{dt} \Big|_{nT} (t-nT)$$

discrete discrete continuous

$$\approx y = C + mx \rightarrow \text{linear in time}$$

linear f's in b/w

dis + effect of gradient



b/w 2 discrete signals,  
now our value is  
opened continuously

Z-transforms : Discrete time systems

Laplace transform<sup>n</sup> to continuous functions.

$$z[y(t)] = \bar{y}(z) = \sum_{n=0}^{\infty} y(nT) z^{-n}$$

$$z\{y(0), y(T), y(2T), \dots\} = \sum_{n=0}^{\infty} y(nT) z^{-n}$$

from now

where  $y^*(s) = \sum_{n=0}^{\infty} y(nT) e^{-nTs}$

When  $(z = e^{Ts})$

$$\bar{y}^*(s) = \sum_{n=0}^{\infty} y(nT) z^{-n} = \bar{y}(s)$$

- all properties of Z-transformation are very similar to L-transform

Format : Laplace

Model  $\rightarrow$  input-output  $\rightarrow$  transform  $f^n \rightarrow y(t) \rightarrow L^{-1}\{g(s)\bar{u}(s)\}$

Format : Z-transform

Model  $\rightarrow$  input-output  $\rightarrow$  transform  $f^n \rightarrow y(t) \rightarrow z^{-1}$

# Z-transform of a unit step $f^n$

Data  
Prog

① Unit step  $f^n : \{1, 1, 1, \dots\}$

$$\bar{z}(u) = 1 \cdot z^0 + 1 \cdot z^{-1} + 1 \cdot z^{-2} + \dots$$

$$= \sum_{n=0}^{\infty} z^{-n}$$

$$\bar{z}(u) = \frac{1}{1-z^{-1}} \quad | \quad L(u) = \frac{A}{s}$$

② Ramp  $f^n : u = \{0, aT, 2aT, 3aT, \dots\}$

$$\begin{aligned} \bar{z}(u) &= 0 \cdot z^0 + aTz^{-1} + 2aTz^{-2} + 3aTz^{-3} + \dots \\ &= aT(1 + 2z^{-1} + 3z^{-2} + \dots)z^{-1} \\ &= \frac{aTz^{-1}}{(1-z^{-1})^2} \end{aligned}$$

Dynamics of a discrete time first-order system:

$$y_{n+1} = ay_n + bu_n$$

Unit step :  $u_n = 1 \quad \forall n$

$$y_{n+1} = ay_n + b$$

$$y_1 = ay_0 + b$$

$$y_2 = ay_1 + b = a(ay_0 + b) + b$$

$$\Rightarrow y_2 = a^2y_0 + (a+1)b$$

$$y_n = a^n y_0 + b(a^{n-1} + a^{n-2} + \dots + 1)$$

Deviation variable at  $t=0$ ,  $y_0 = 0$

$$y_n = b(a^{n-1} + a^{n-2} + \dots + 1)$$

$$y_n = b \left[ \frac{1-a^n}{1-a} \right]$$

local

a 1.1  
b 1.1

y<sub>n</sub>

$a = 0.1$

$a = 1.1$

$a = 0.9$

Date \_\_\_\_\_

Page \_\_\_\_\_

Imp.  $|0|: |a < 1|$  stable  
 $|a > 1|$  unstable

do w/ z-transform

$$g(z) = \frac{\bar{y}(z)}{\bar{u}(z)}$$

$$\Rightarrow \bar{y}(z) = \bar{u}(z)g(z)$$

$$\bar{u}(z) = \frac{1}{1-z^{-1}}$$

$g(z)$  unknown

$$g(s) = \frac{K}{s+1}$$

$$g(t) = \frac{K}{\tau} e^{-t/\tau}$$

$$g(z) = \sum_{n=0}^{\infty} \frac{K}{\tau} e^{-nT/\tau} z^{-n}$$

$$= \left( \frac{K}{\tau} \right) \cdot \frac{1}{(1 - e^{-nT/\tau}) z^{-1}}$$

AB

$$g(z) = \frac{b}{1 - az^{-1}}$$

$$y(z) = \left( \frac{b}{1 - az^{-1}} \right) \left( \frac{1}{1 - z^{-1}} \right)$$

Tables -

inverse :

$$y(n) = b \left( \frac{1 - a^n}{1 - a} \right)$$