

Finite Element Method: A Homemade 2D Solver

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<https://github.com/vtereshkov/fem>

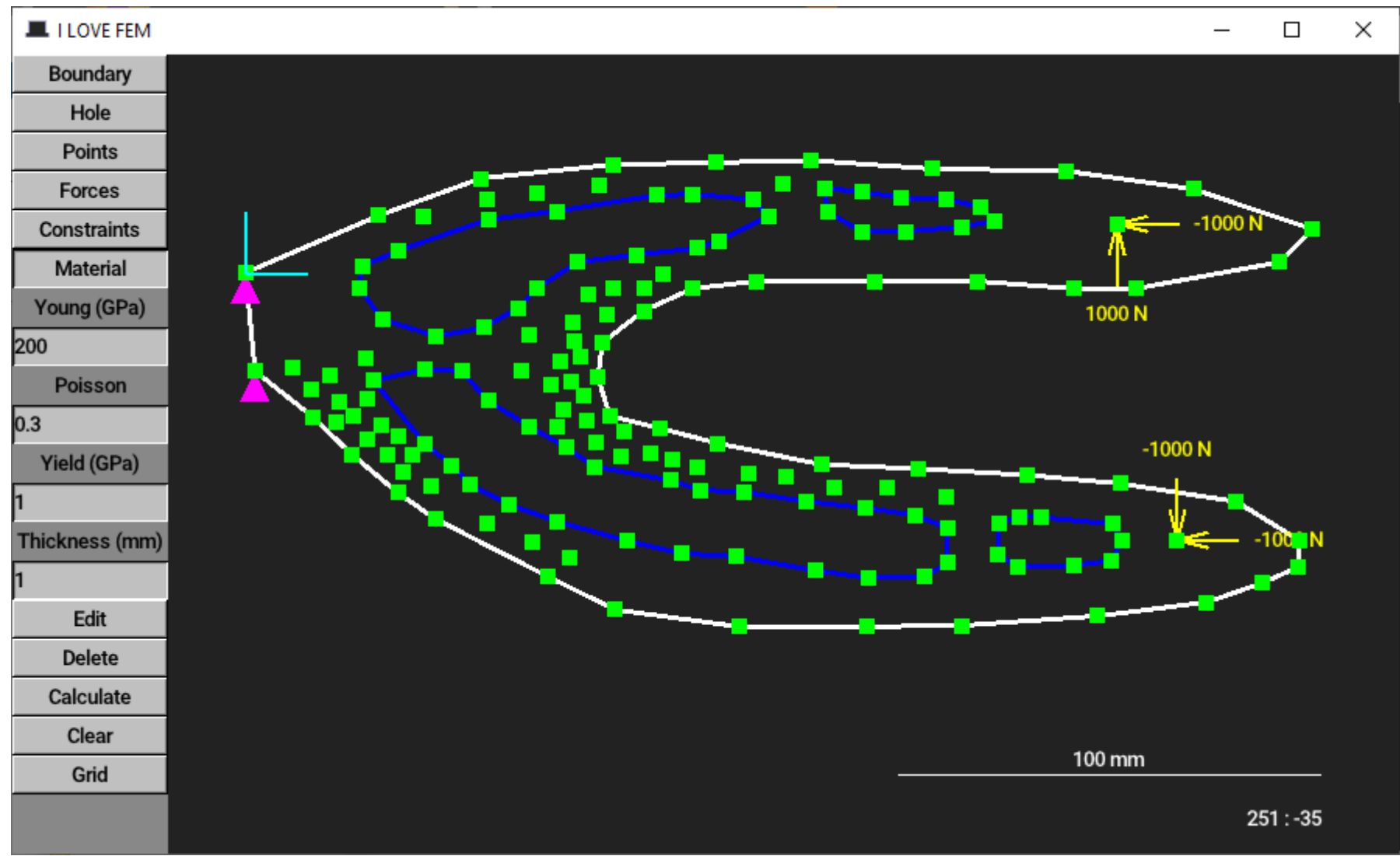
Topics

- Editor
- Mesh generation
- FEM solver
- Visualization
- Self-check
- Implementation

Editor: Requirements

- **2D plate boundary:** one non-self-intersecting polygon
- **Holes:** only polygons, no circles
- **Extra points** for manual mesh refinement
- **Constraints:** each fixing a point in both X and Y
- **External loads:** only forces, no torques
- **Material:** Young modulus, Poisson ratio, max stress

Editor: GUI



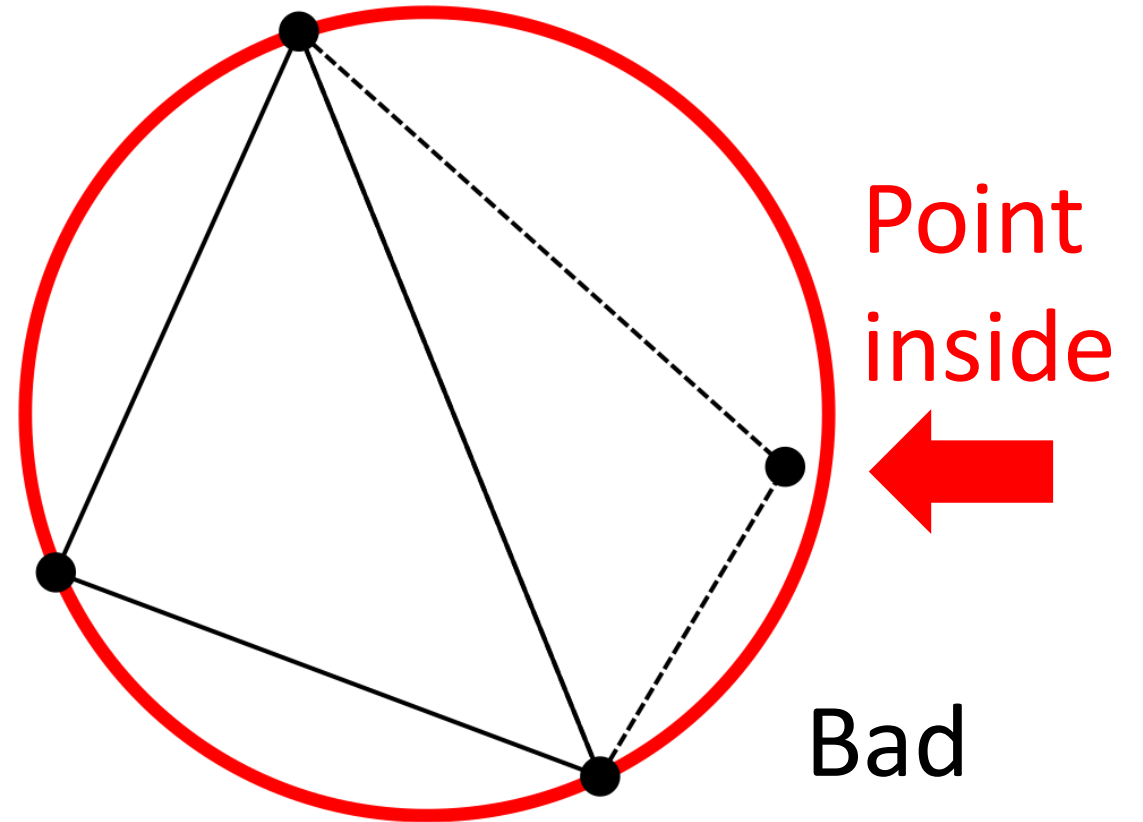
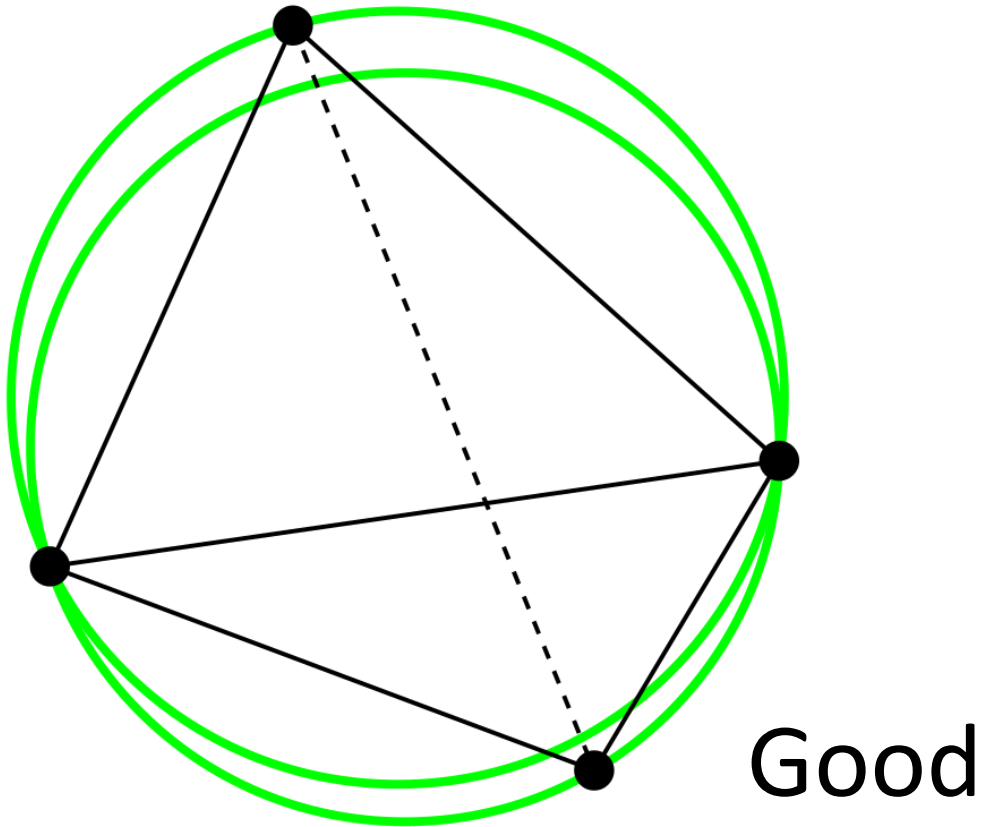
Editor: Clockwise or counter-clockwise?

- **Boundary:** counter-clockwise
- **Holes:** clockwise

```
fn (ed: ^Editor) isClockwise(poly: Poly): bool {  
    sum := 0.0  
    for i, pt1 := 0, ed.pts[poly.vert[len(poly.vert) - 1]]; i < len(poly.vert); i++ {  
        pt2 := ed.pts[poly.vert[i]]  
        sum += (pt2.x - pt1.x) * (pt2.y + pt1.y)  
        pt1 = pt2  
    }  
    return sum > 0.0  
}
```

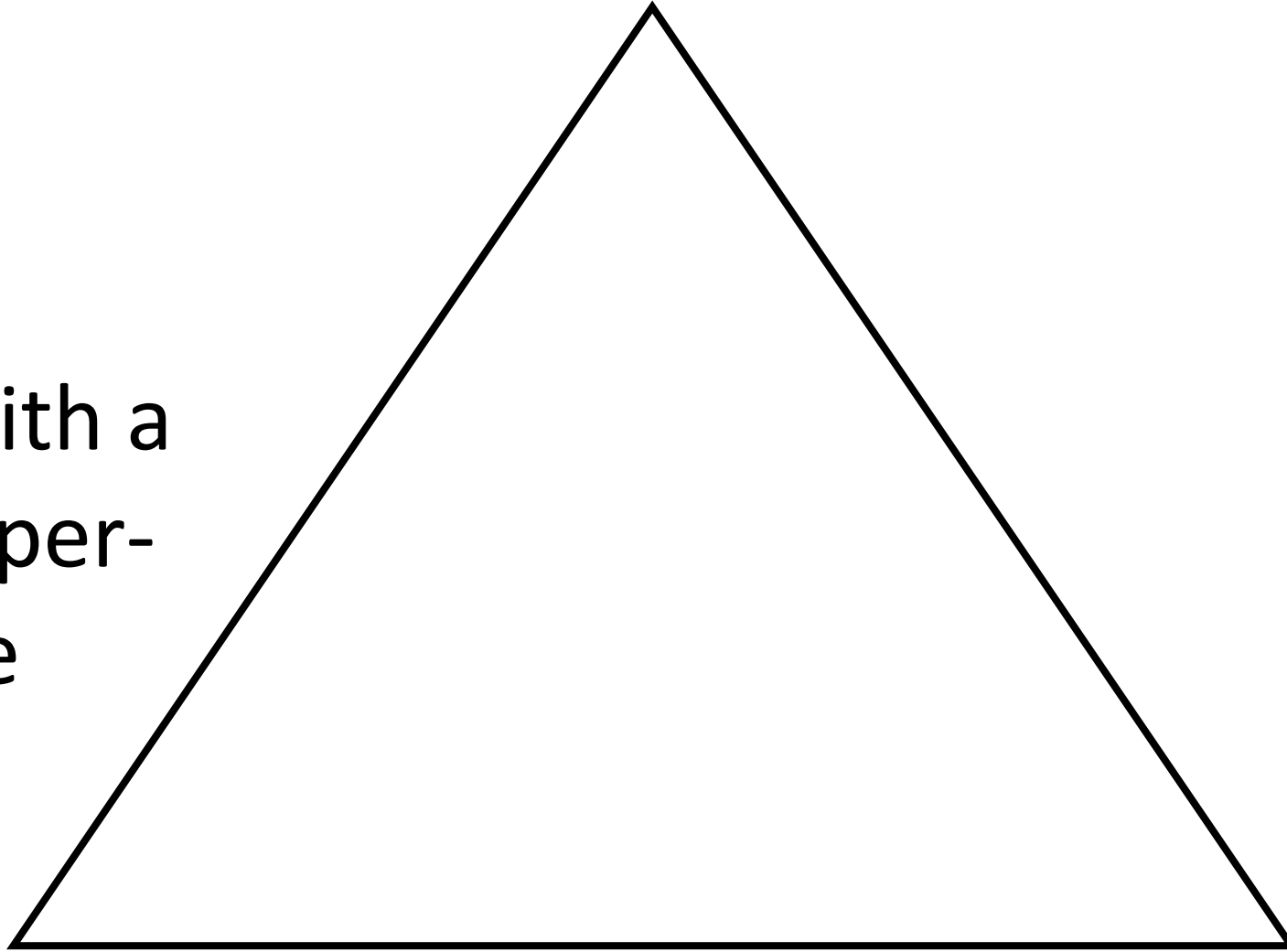
Mesh generation: Delaunay triangulation

How to avoid extremely long and thin triangles?



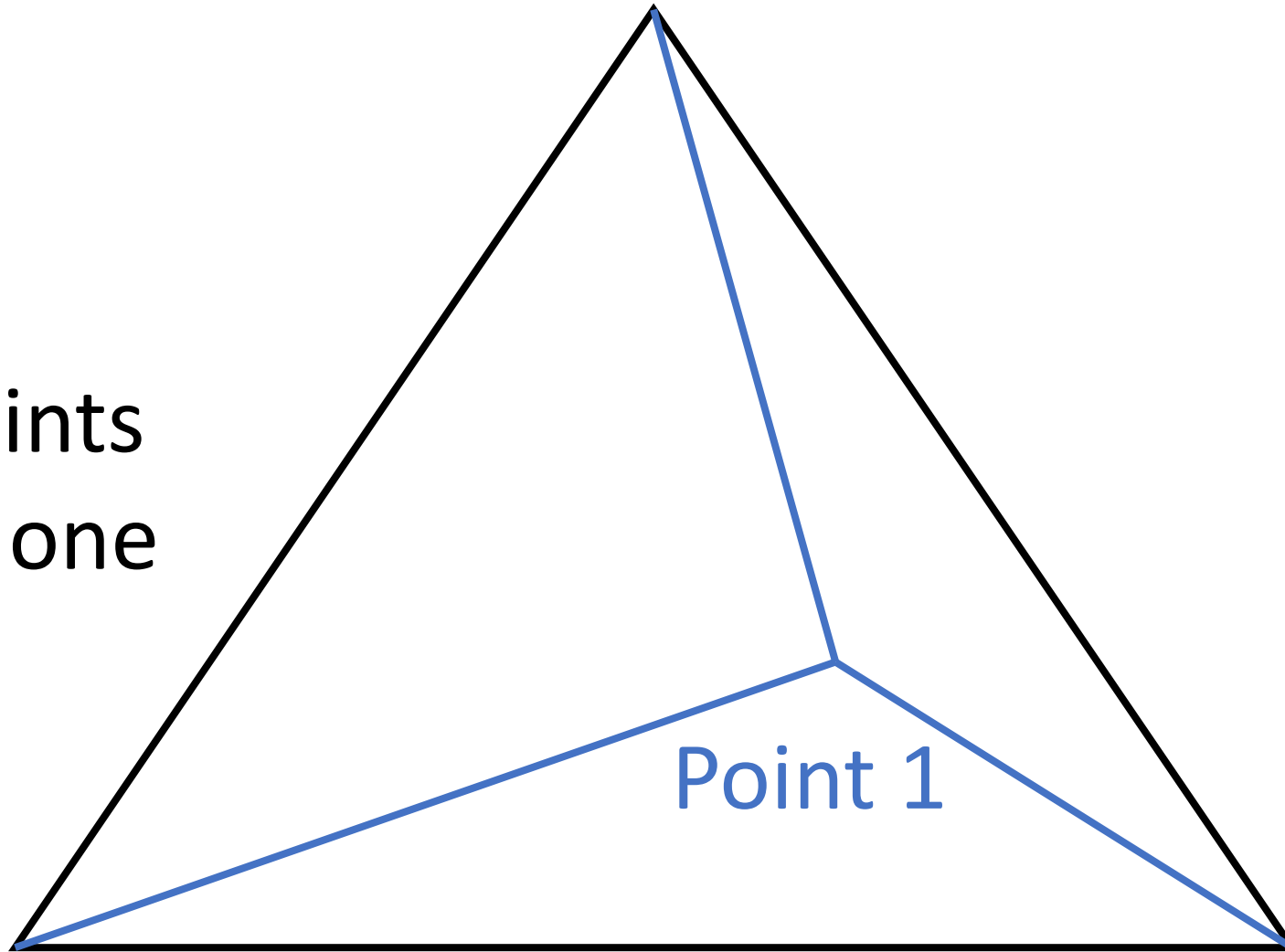
Mesh generation: Bowyer-Watson algorithm

Start with a
fake super-
triangle



Mesh generation: Bowyer-Watson algorithm

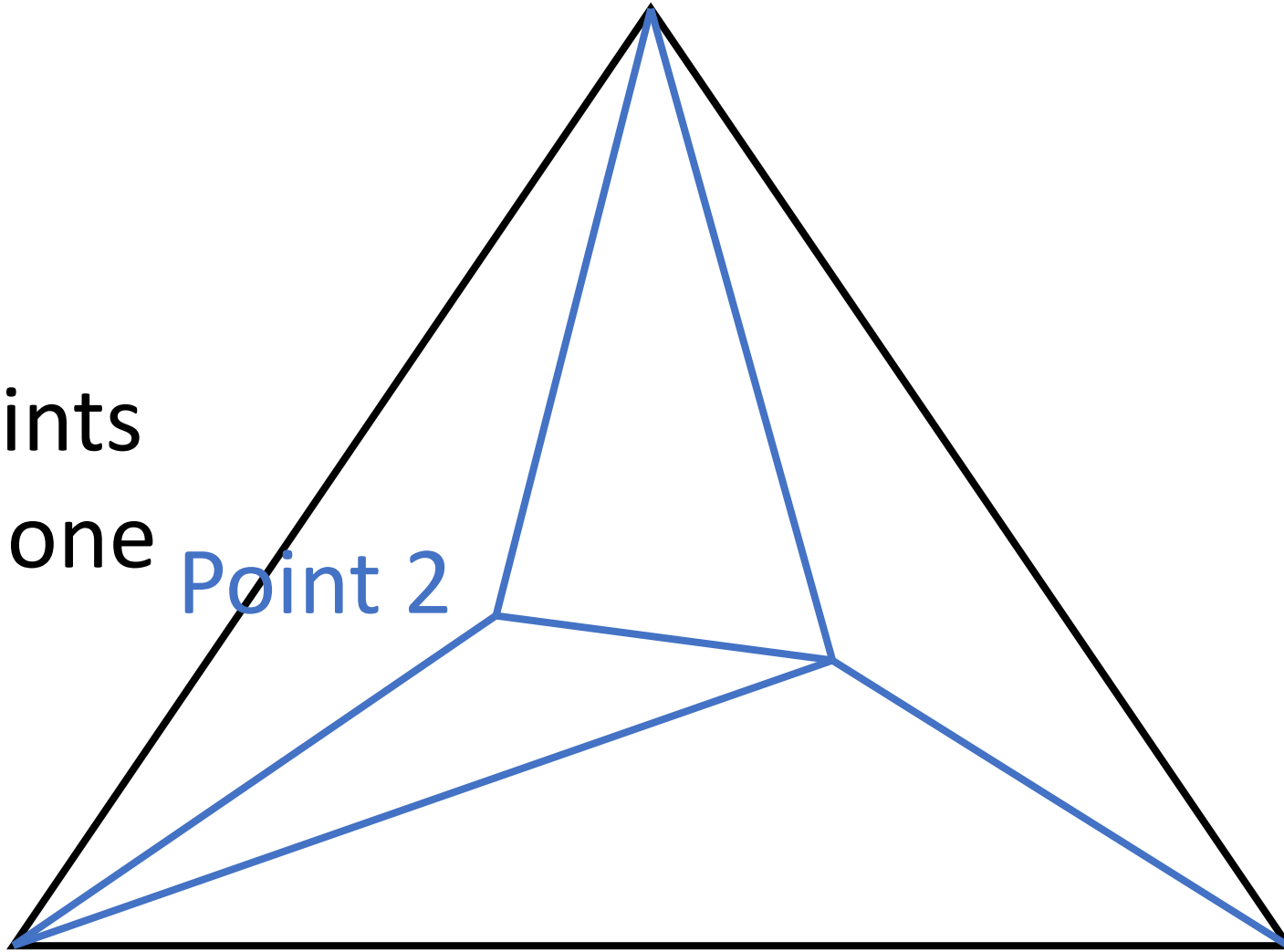
Add points
one by one



Mesh generation: Bowyer-Watson algorithm

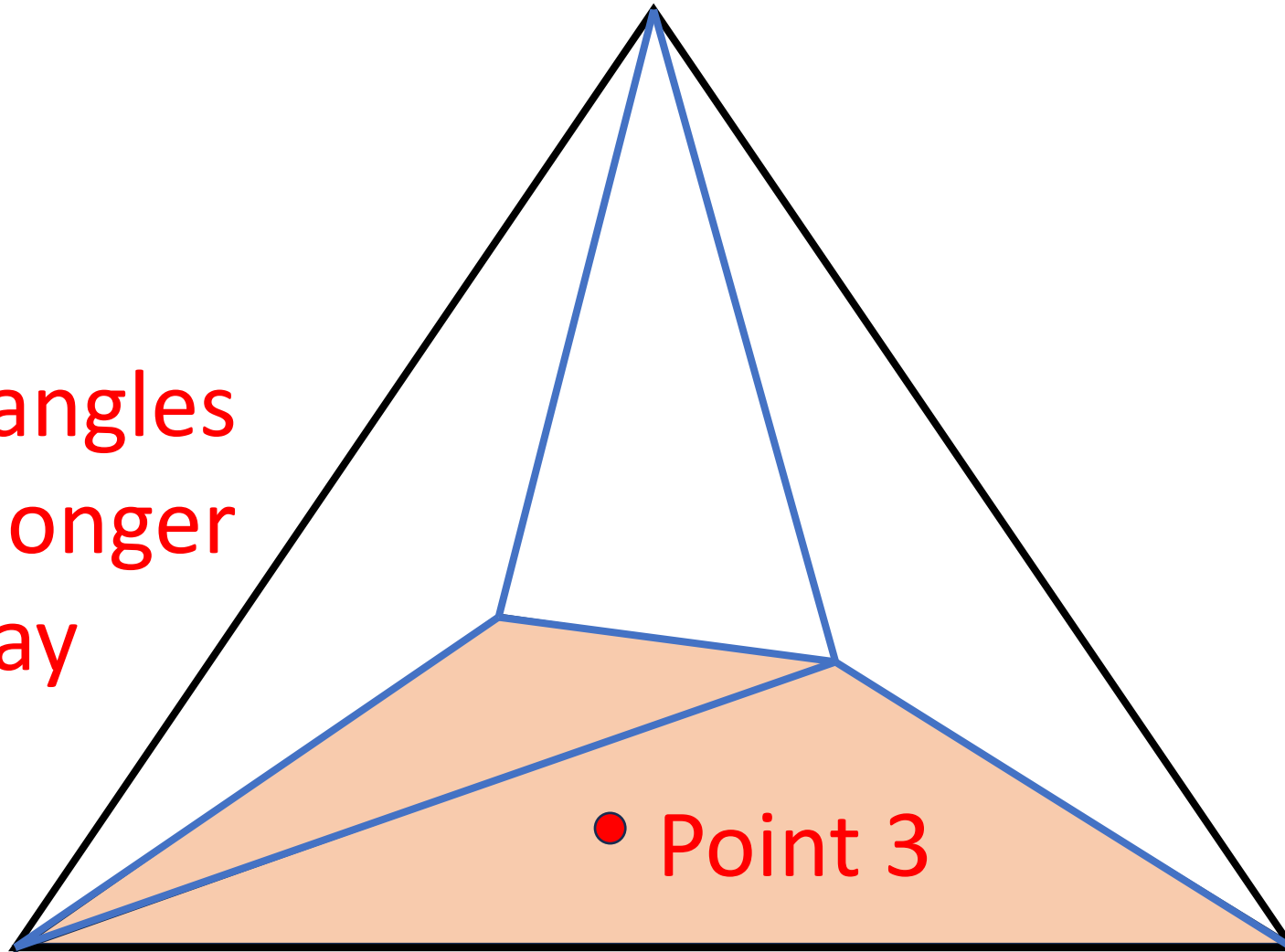
Add points
one by one

Point 2



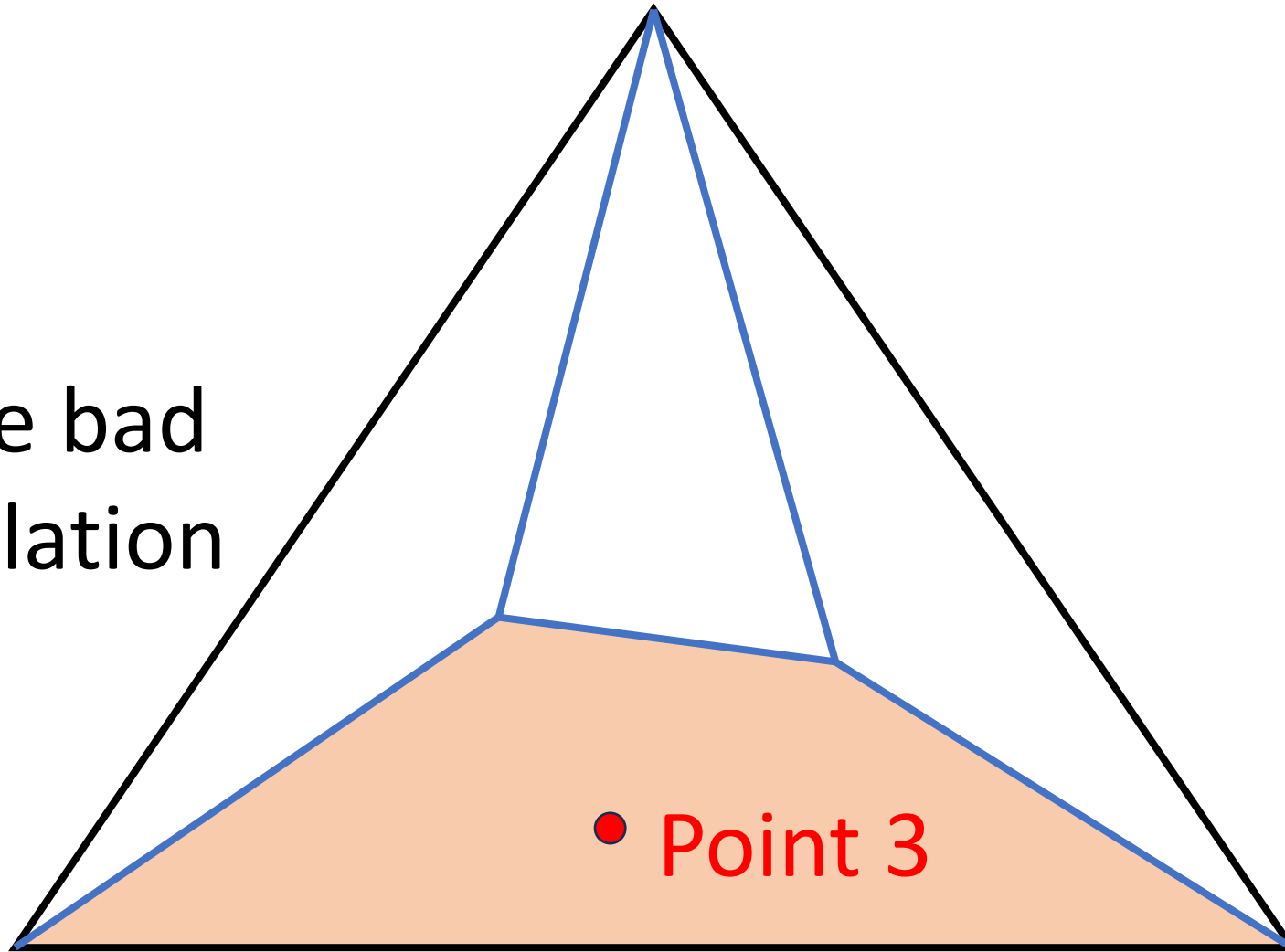
Mesh generation: Bowyer-Watson algorithm

Two triangles
are no longer
Delaunay



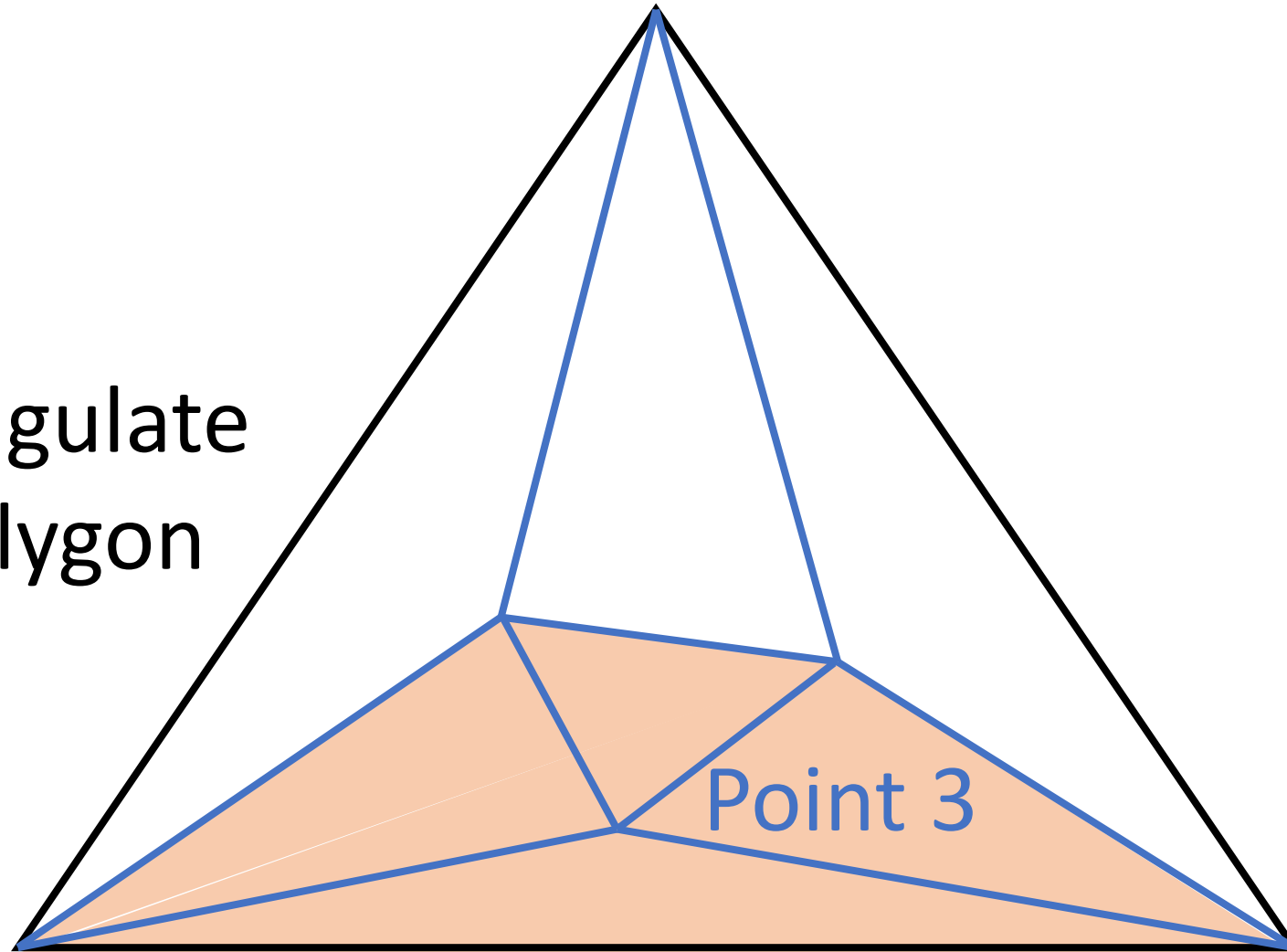
Mesh generation: Bowyer-Watson algorithm

Remove bad
triangulation



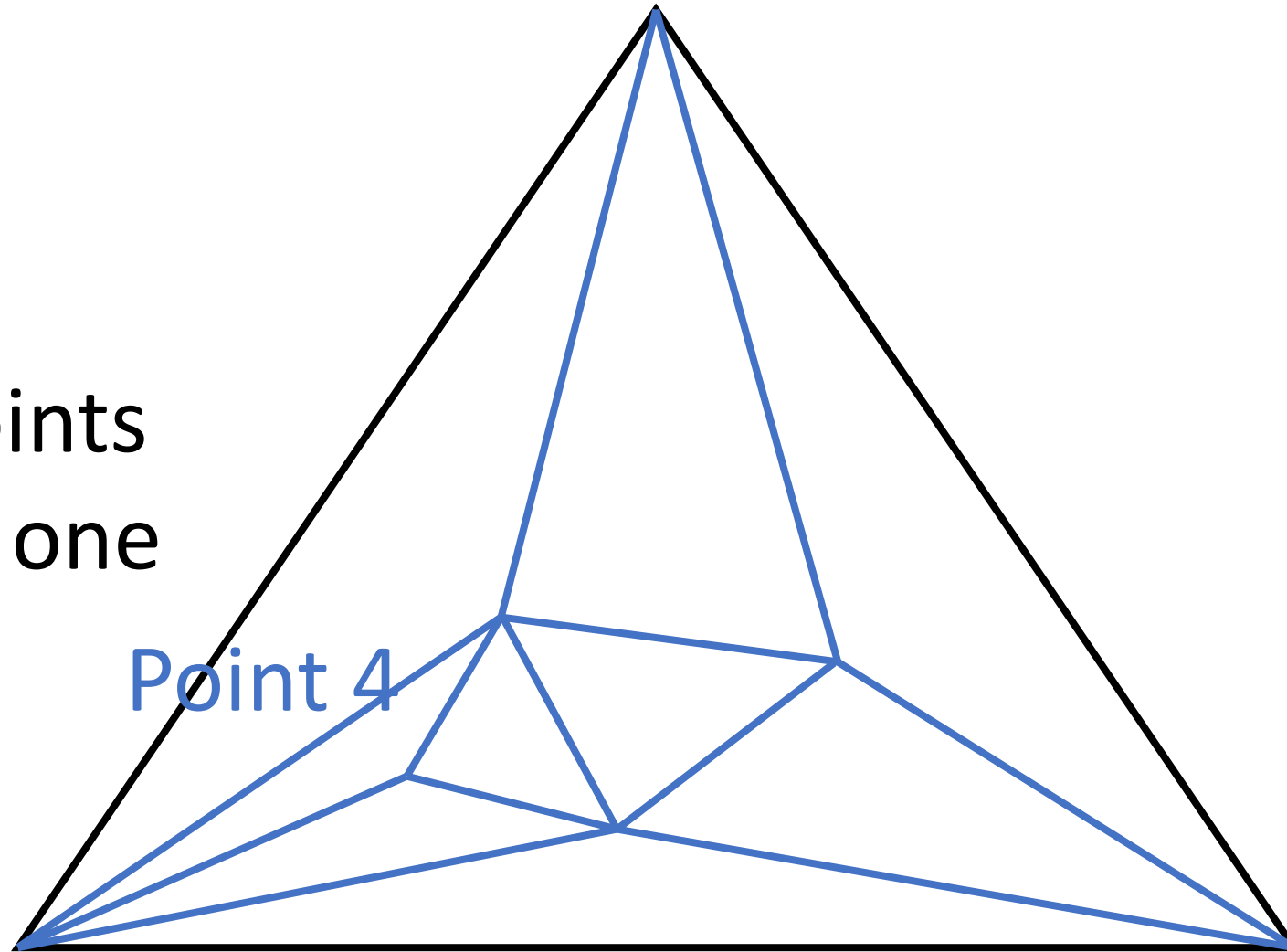
Mesh generation: Bowyer-Watson algorithm

Retriangulate
bad polygon



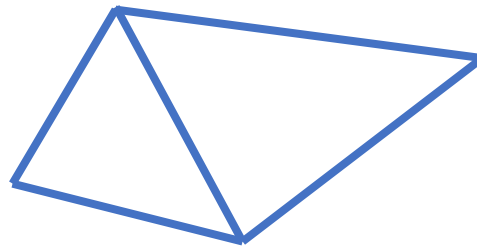
Mesh generation: Bowyer-Watson algorithm

Add points
one by one

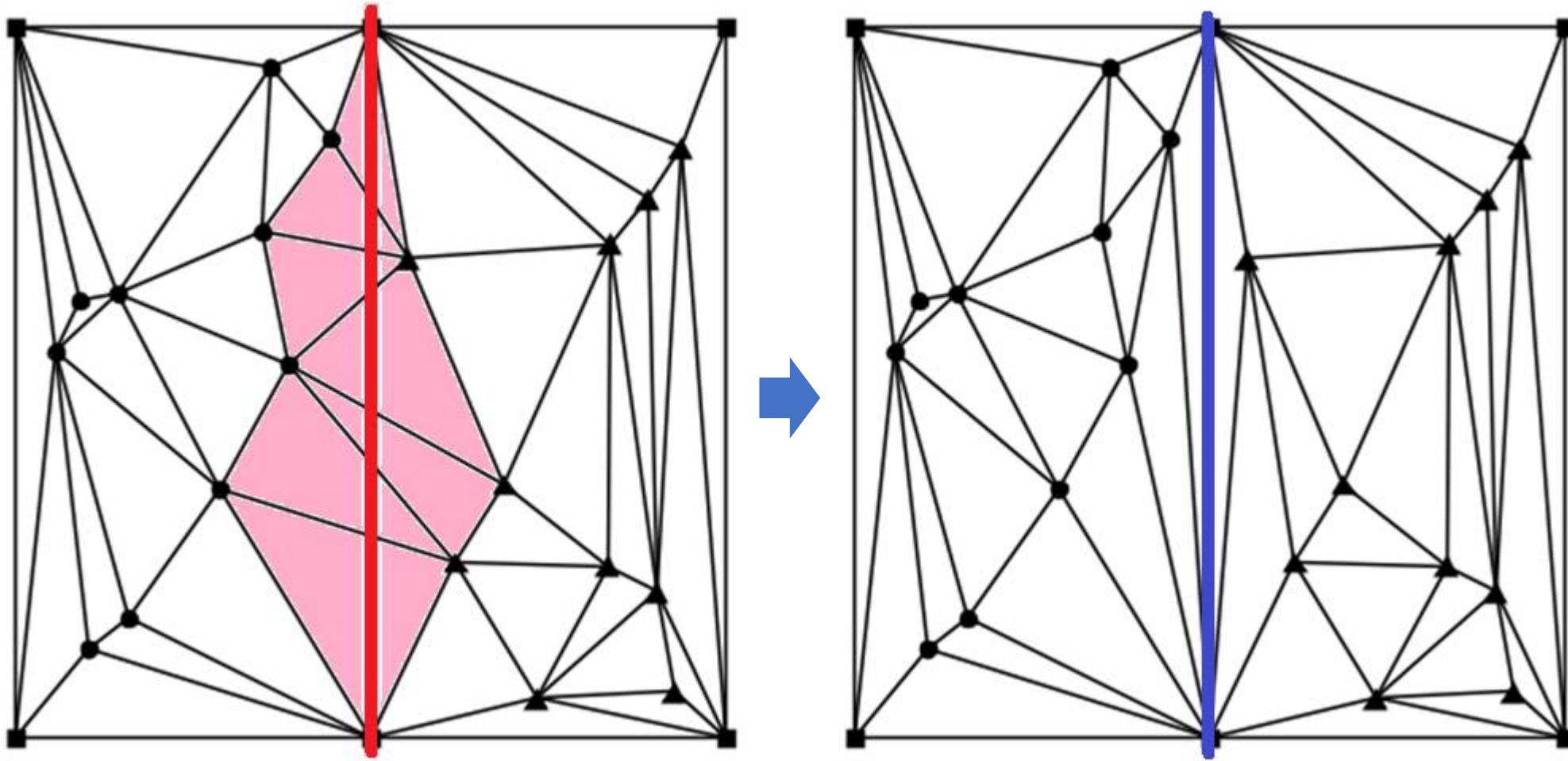


Mesh generation: Bowyer-Watson algorithm

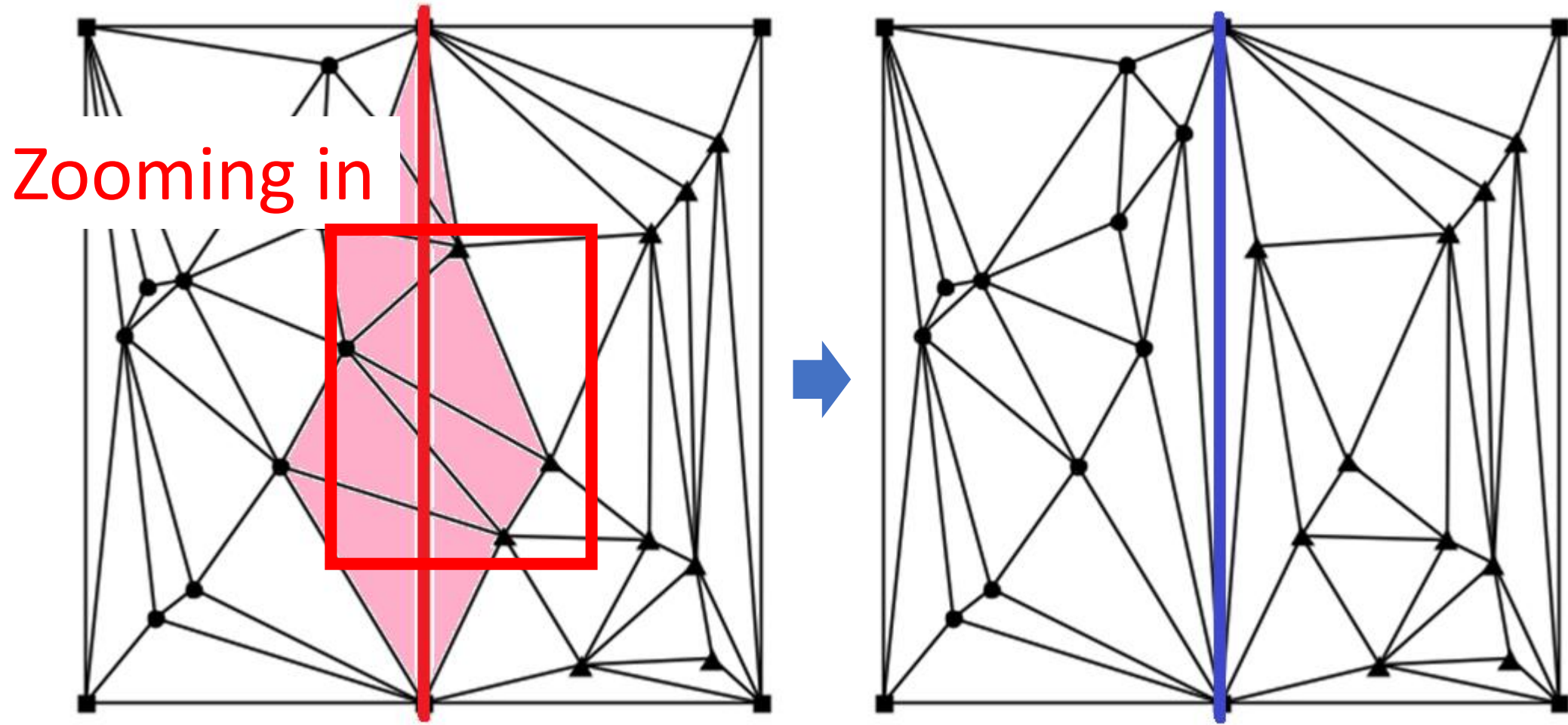
Remove the
super-triangle



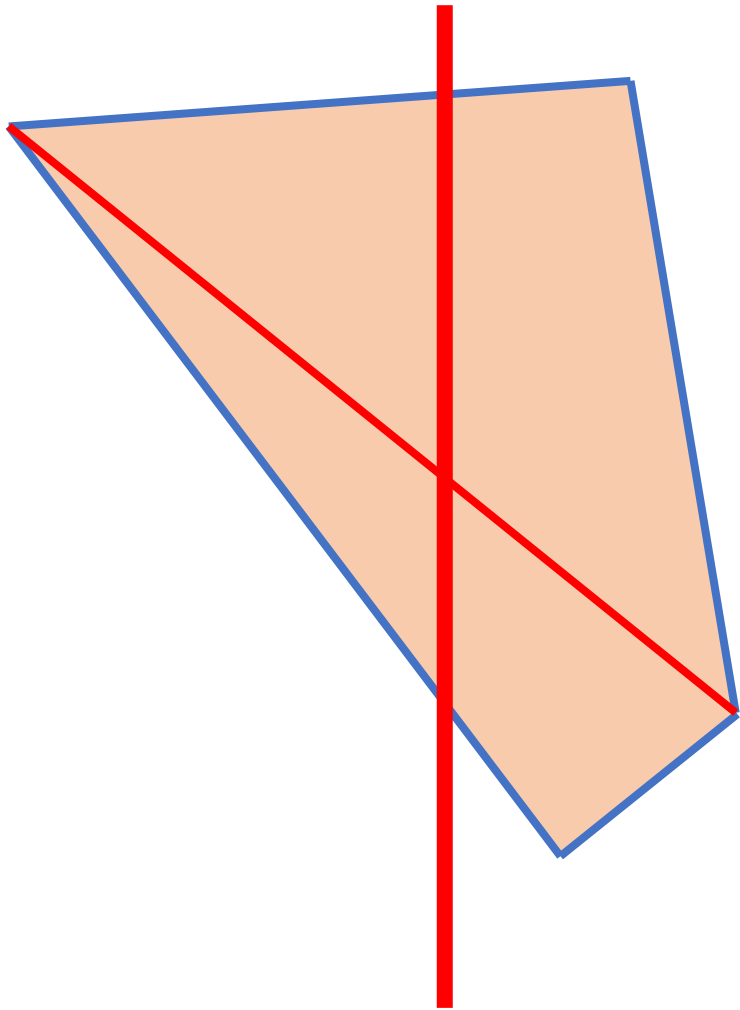
Mesh generation: Missing edges



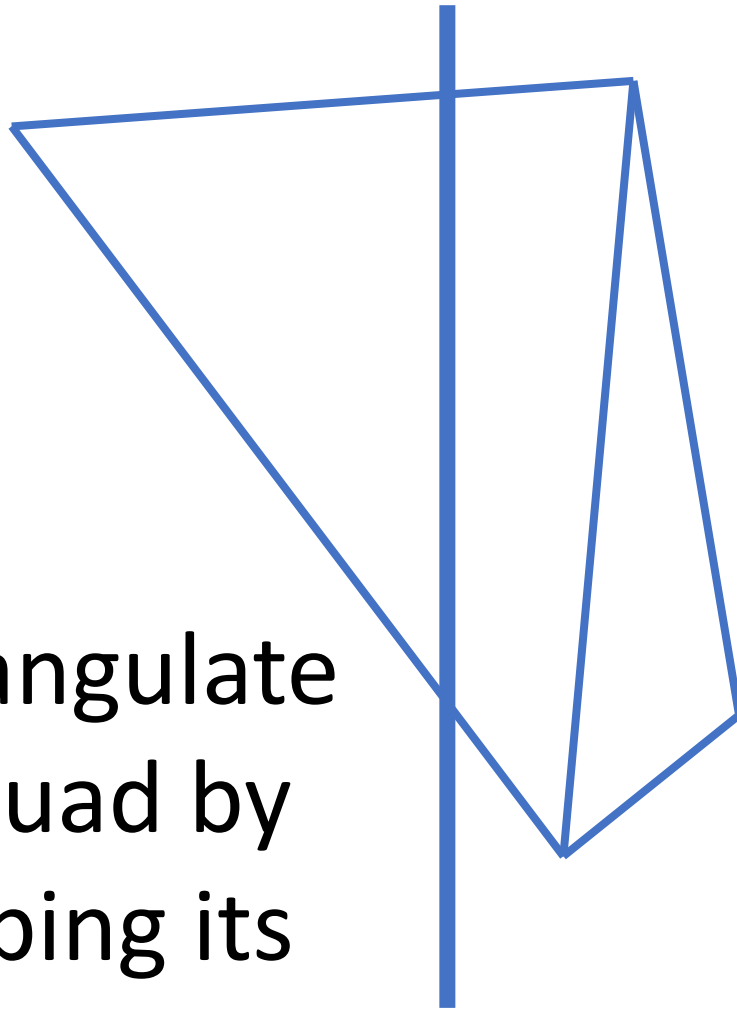
Mesh generation: Missing edges



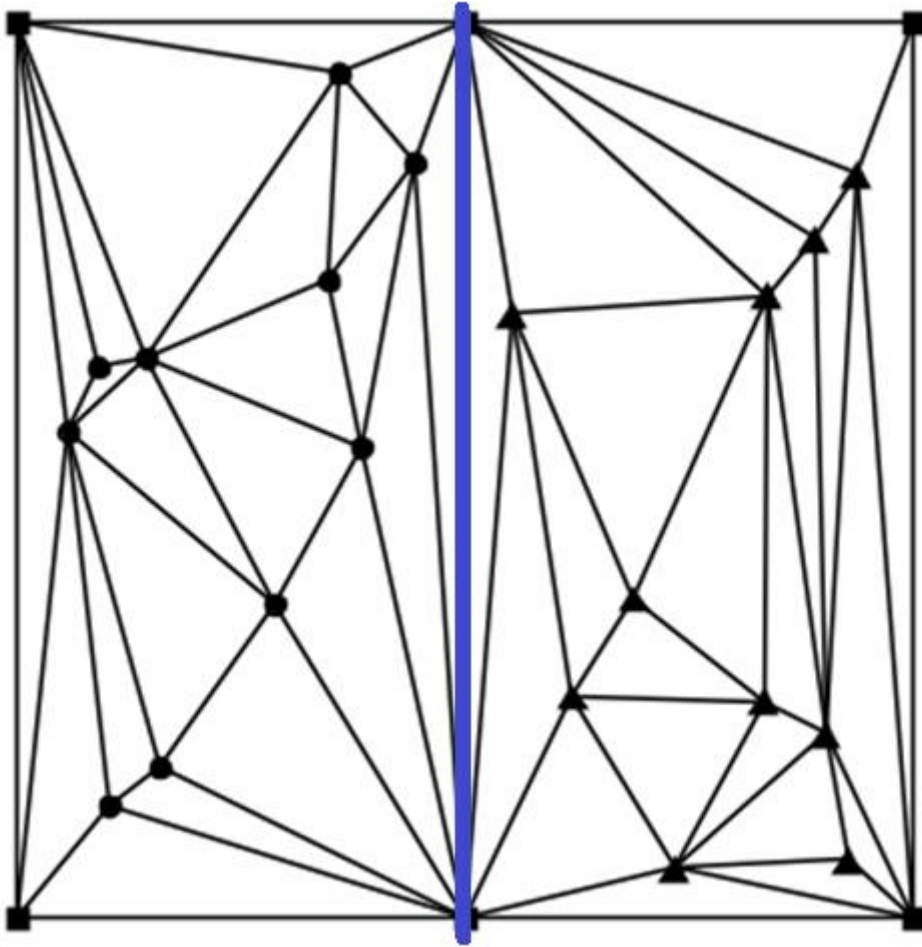
Mesh generation: Missing edges: Sloan algorithm



Retriangulate
bad quad by
swapping its
diagonals

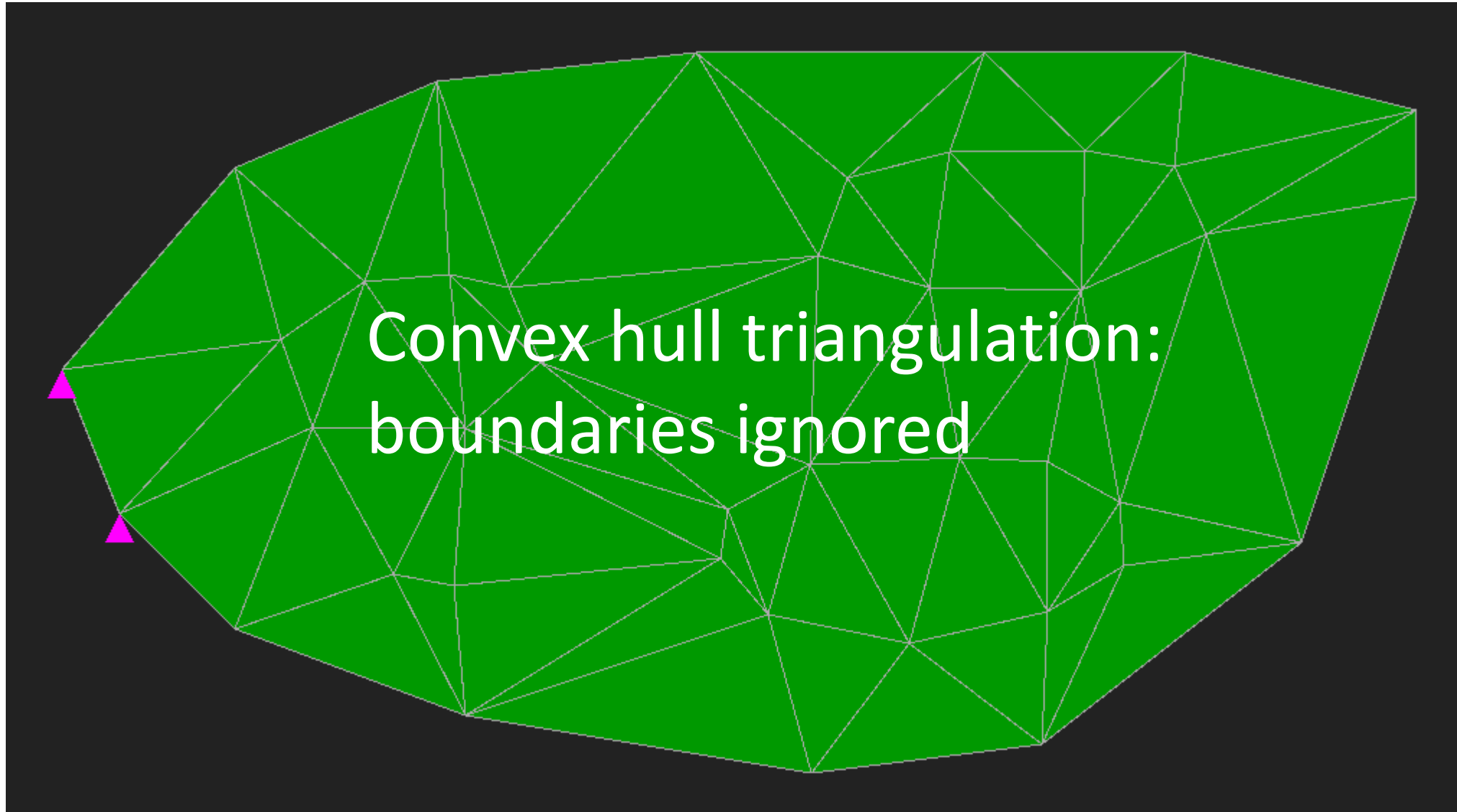


Mesh generation: Missing edges

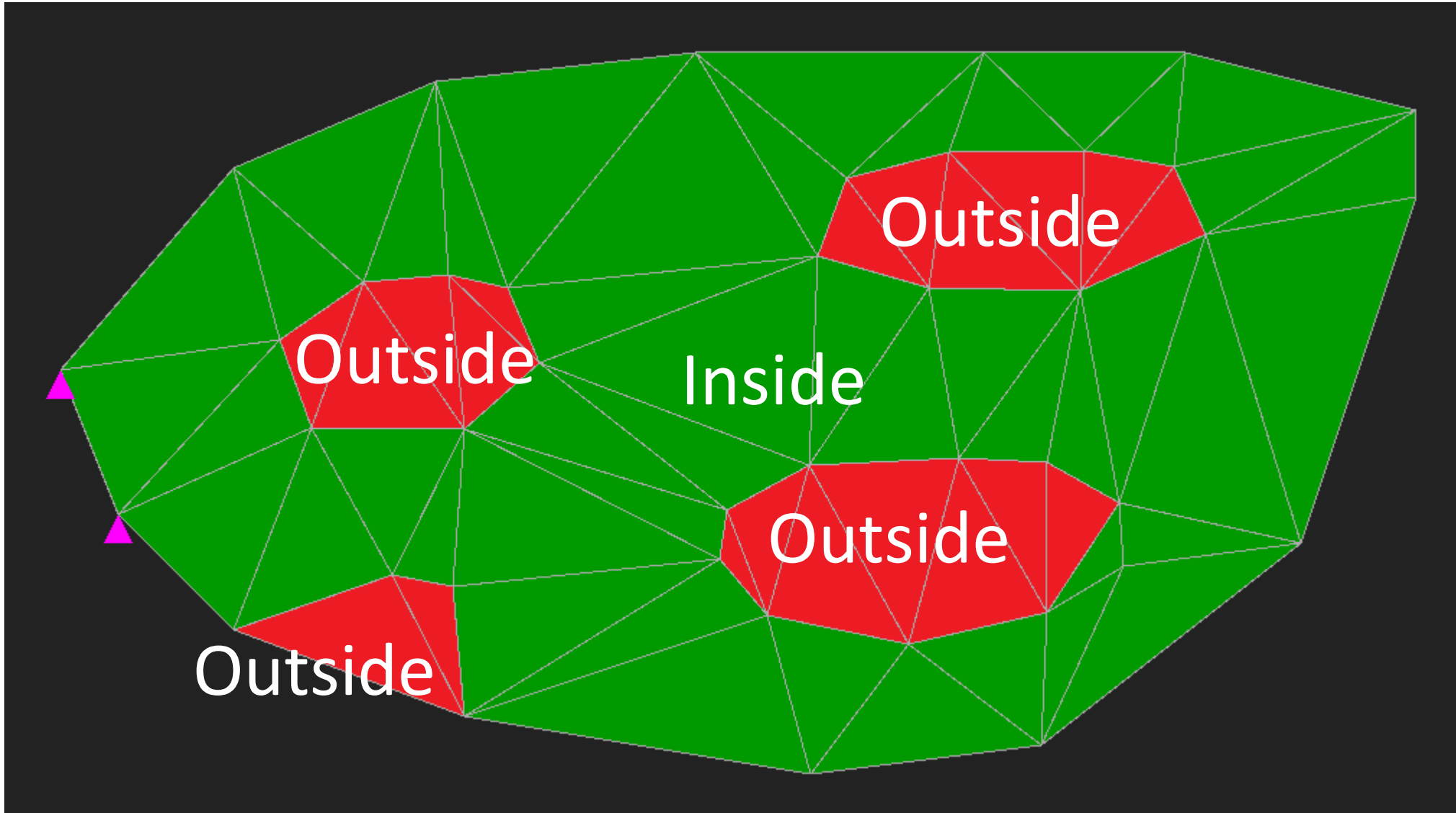


No longer
Delaunay, but
who cares if we
need this edge?

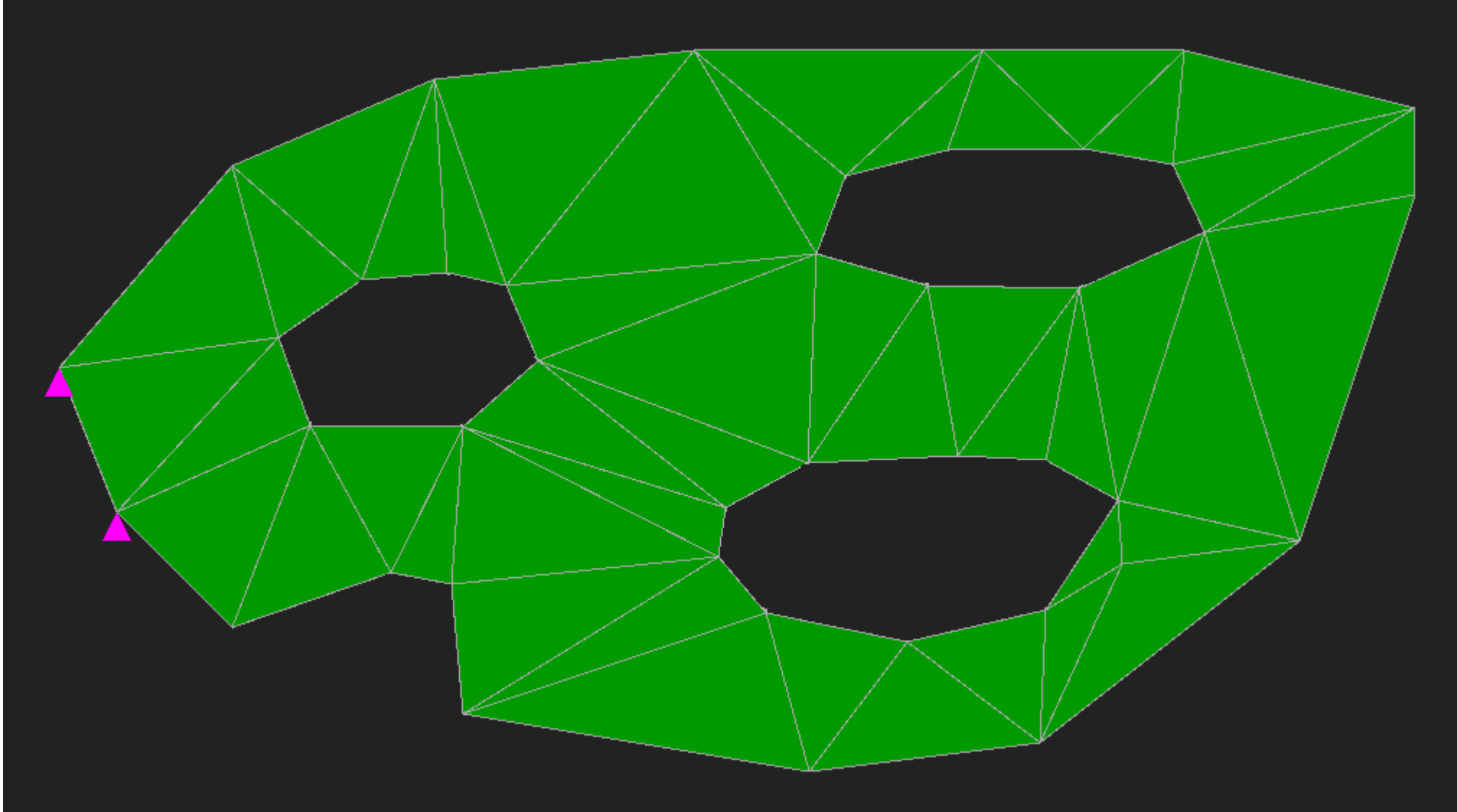
Mesh generation: Removing outside triangles



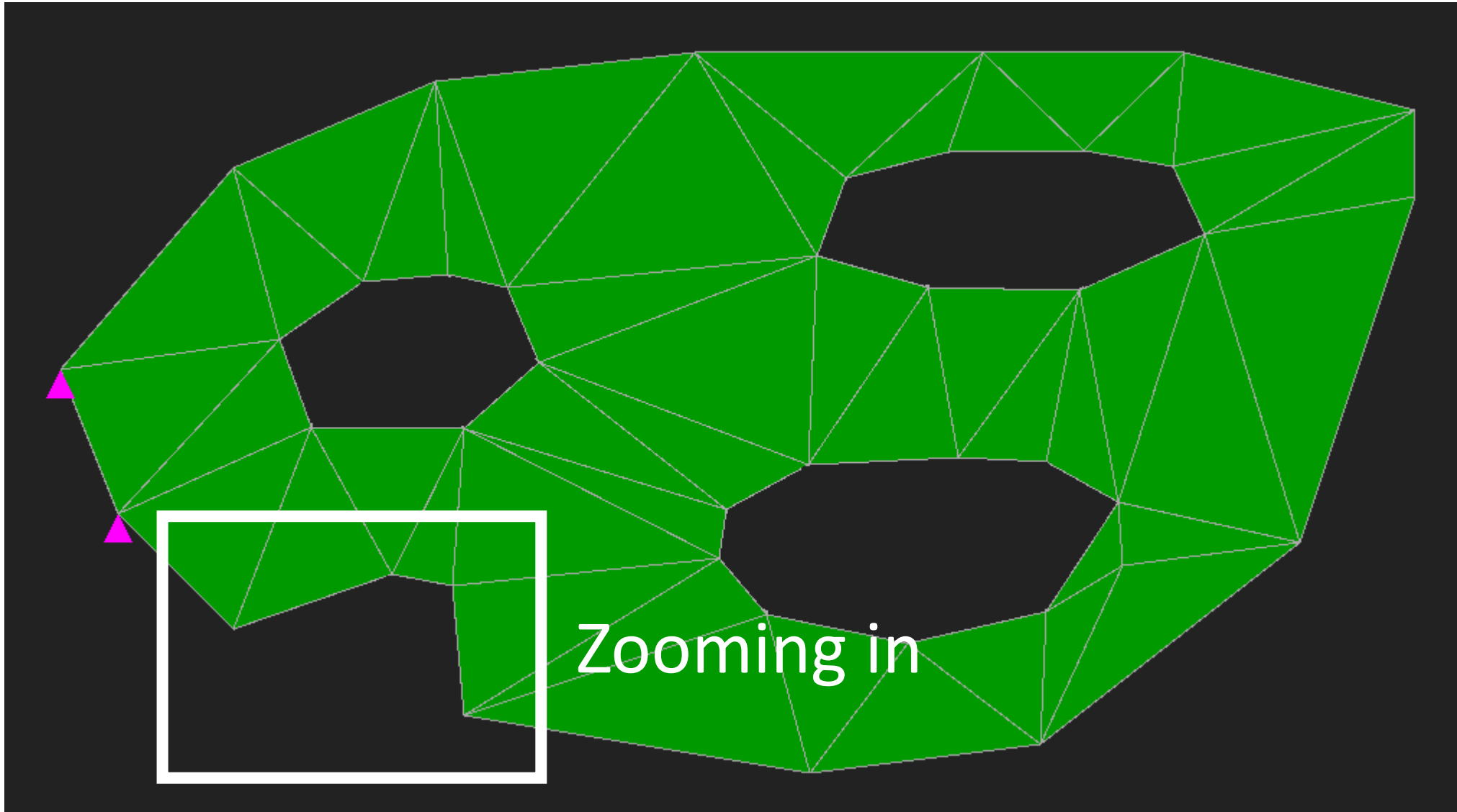
Mesh generation: Removing outside triangles



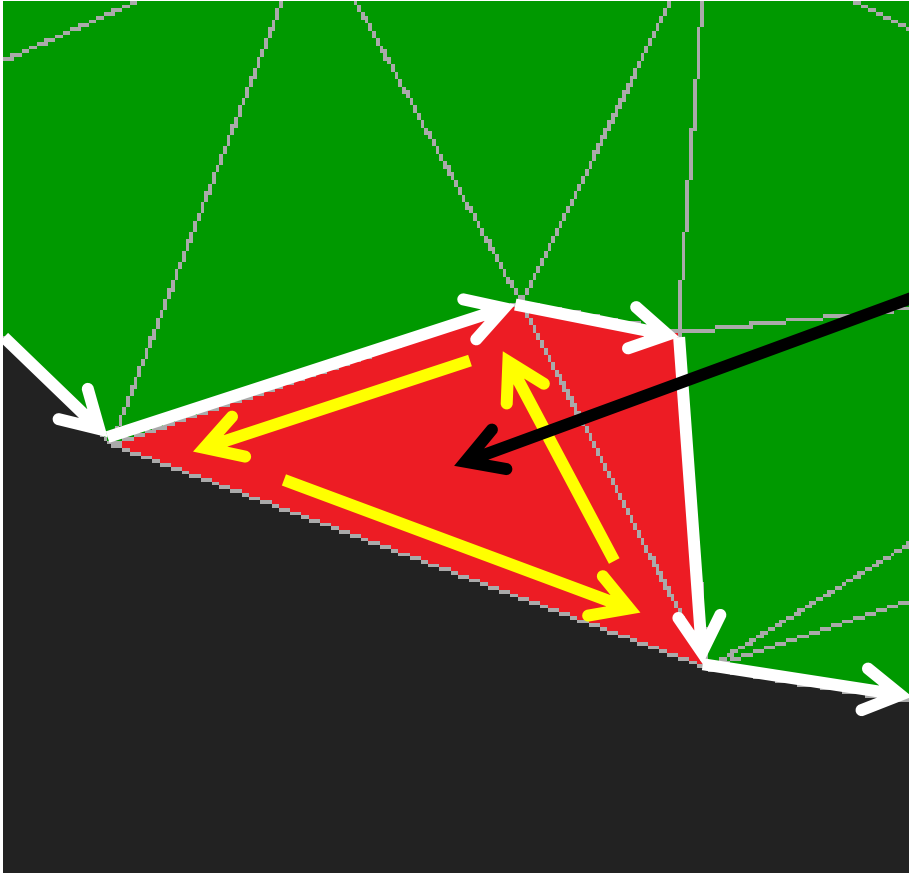
Mesh generation: Removing outside triangles



Mesh generation: Removing outside triangles

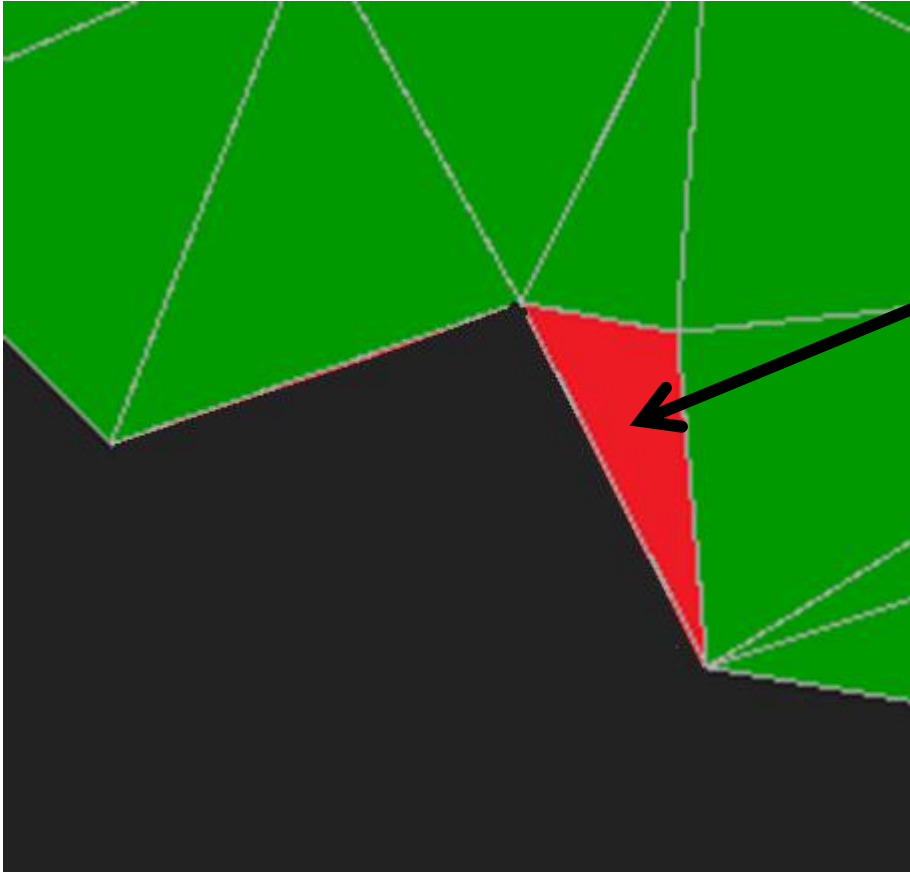


Mesh generation: Removing outside triangles



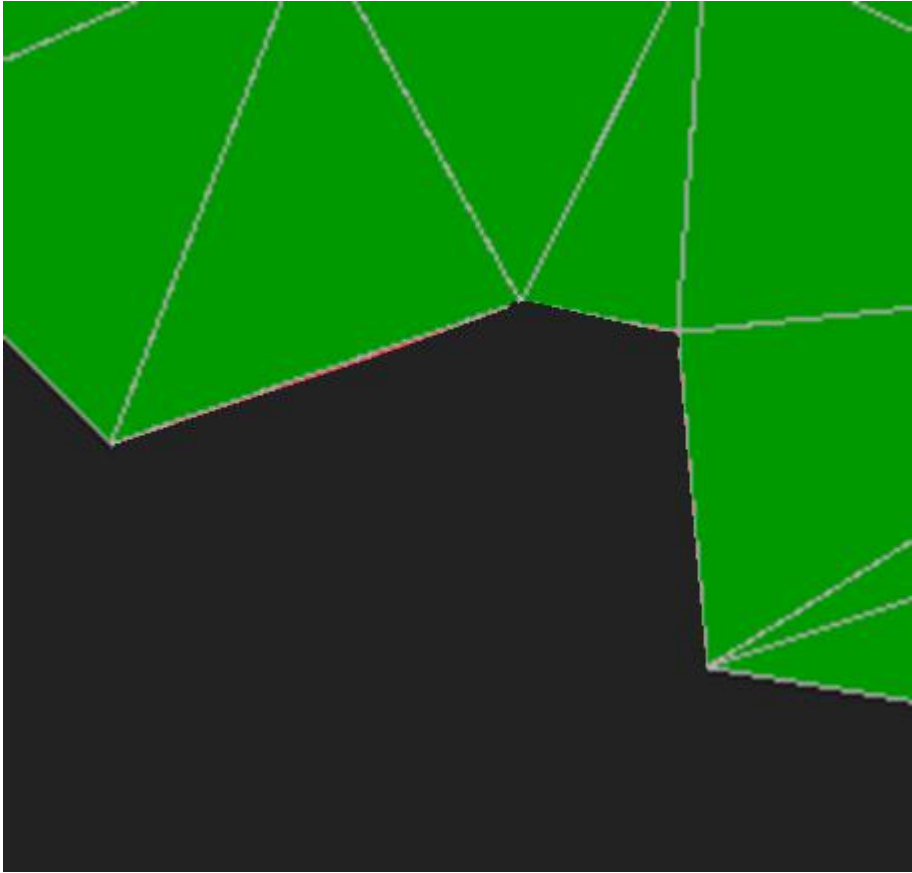
First outside triangle:
contains a boundary
segment, but in opposite
direction

Mesh generation: Removing outside triangles



Next outside triangle(s):
all neighbors, except the
inside triangles

Mesh generation: Removing outside triangles



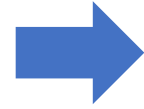
Remove recursively!

Mesh generation: Validation

- **Boundary:**
 - No intersections
 - Each point must belong to 0 or 2 edges
- **Mesh:**
 - Each point must belong to a triangle

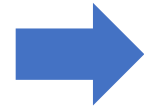
FEM: Discretization

∞ points



N points (nodes)

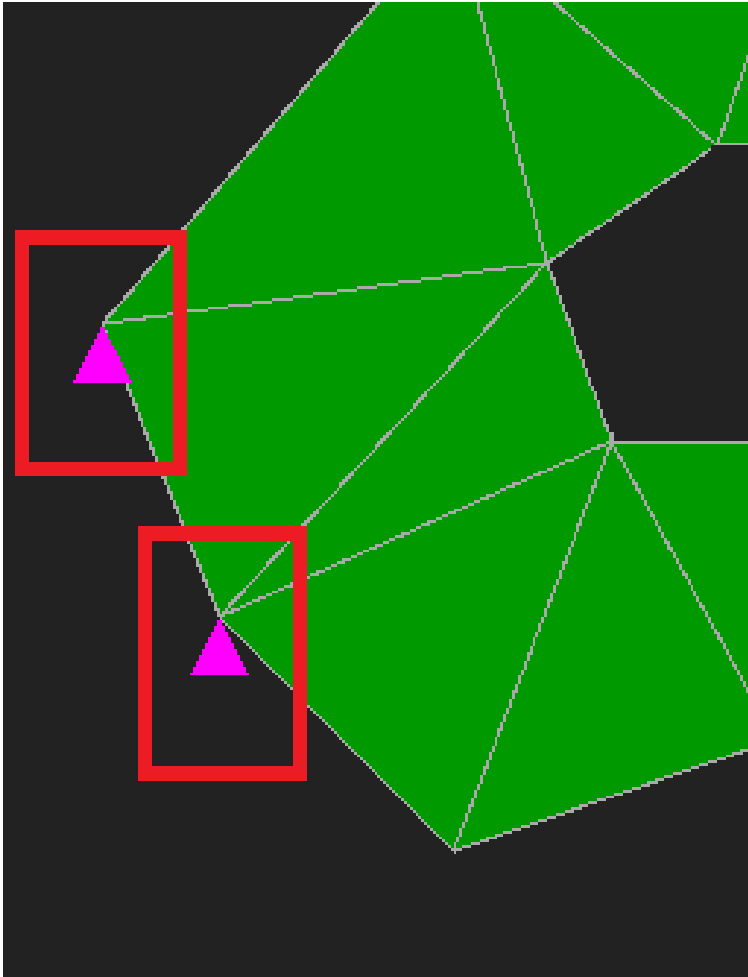
∞ unknown
displacements



$2N$ unknown
displacements

Interpolate in
between

FEM: Constraints



No displacement



No DOF



No equation

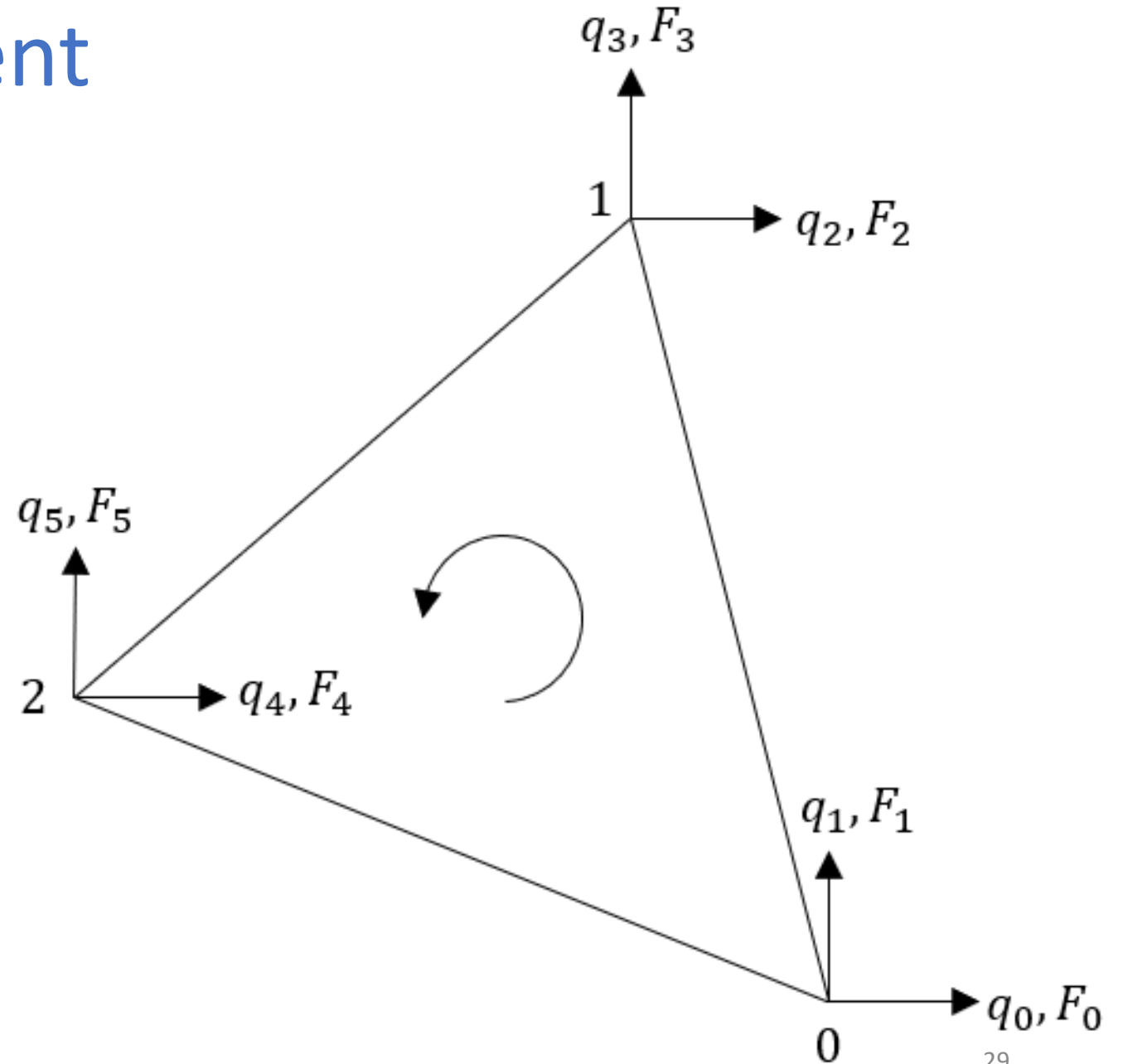


No reaction force

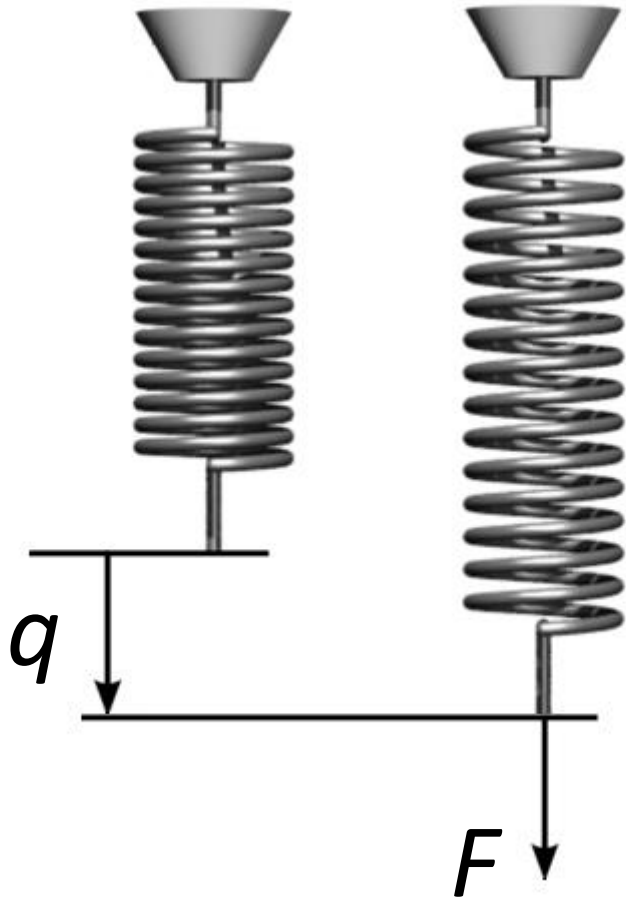
FEM: Triangular element

q = unknown
independent
displacements
(DOFs)

F = external forces
(0 if not set)



FEM: Triangular element: “Hooke’s law”



$$q \longleftrightarrow \overset{?}{F}$$

1 DOF: $kq = F$

6 DOFs:
$$k \begin{bmatrix} q_0 \\ \vdots \\ q_5 \end{bmatrix} = \begin{bmatrix} F_0 \\ \vdots \\ F_5 \end{bmatrix}$$

FEM: Triangular element: “Hooke’s law”

In scalar form:

$$k_{00}q_0 + k_{01}q_1 + \cdots + k_{05}q_5 = F_0$$

$$\vdots$$

$$k_{50}q_0 + k_{51}q_1 + \cdots + k_{55}q_5 = F_5$$

FEM: Triangular element: Stiffness matrix

$$k = \begin{bmatrix} k_{00} & \dots & k_{05} \\ \vdots & \vdots & \vdots \\ k_{50} & \dots & k_{55} \end{bmatrix}$$

- 6×6 matrix
- Indexed by element DOFs
- Depends on triangle sides, material and thickness

k_{ij} = force F_i needed to make $q_j = 1$

Explicit formula in `fem::createElement()`

FEM: Triangular element: Stiffness matrix

```
det := b.x * c.y - c.x * b.y  
area := 0.5 * fabs(det)
```

```
el.geometry = matrix::Matrix{  
    {-b.y,    0, -c.y,    0, -a.y,    0},  
    {  0,  b.x,    0,  c.x,    0,  a.x},  
    { b.x, -b.y,  c.x, -c.y,  a.x, -a.y}  
}.mulf(1.0 / det)
```

```
el.elasticity = matrix::Matrix{  
    {          1, material.poisson,          0},  
    {material.poisson,          1,          0},  
    {          0,          0, (1 - material.poisson) / 2}  
}.mulf(material.young / (1 - material.poisson * material.poisson))
```

```
el.stiffness = el.geometry.transpose().mul(el.elasticity).mul(el.geometry)  
    .mulf(material.thickness * area)
```

FEM: Connecting elements

- **Element:** 6 DOFs
- **Whole model:** $2N$ DOFs

Element DOF
(0...5)



Global DOF
(0... $2N-1$)

FEM: Global stiffness matrix

$$K \begin{bmatrix} q_0 \\ \vdots \\ q_{2N-1} \end{bmatrix} = \begin{bmatrix} F_0 \\ \vdots \\ F_{2N-1} \end{bmatrix}$$

- $2N \times 2N$ matrix
- Indexed by global DOFs
- Made of element stiffness matrices

$$K = \begin{bmatrix} K_{00} & \dots & K_{0,2N-1} \\ \vdots & \vdots & \vdots \\ K_{2N-1,0} & \dots & K_{2N-1,2N-1} \end{bmatrix}$$

FEM: Global stiffness matrix

Pseudocode:

$$K = 0$$

for each *element*

for each *i*

for each *j*

$$K_{mn} += k_{ij}$$

Element DOF

i

j



Global DOF

m

n

FEM: Global stiffness matrix

```
fn getGlobalStiffness(els: []Element, numPts: int): matrix::Matrix {  
    stiffness := matrix::zeros(2 * numPts, 2 * numPts)  
    for _, el in els {  
        for row in el.stiffness {  
            globalRow := el.convertDofToGlobal(row)  
            for col in el.stiffness[0] {  
                globalCol := el.convertDofToGlobal(col)  
                stiffness[globalRow][globalCol] += el.stiffness[row][col]  
            }  
        }  
    }  
    return stiffness  
}
```

FEM: Constraints

Constrained DOFs? Just strike them out!

Constrained DOF

$$K = \begin{bmatrix} K_{00} & K_{01} & \dots & K_{0,2N-1} \\ K_{10} & K_{11} & \dots & K_{1,2N-1} \\ \vdots & \vdots & \vdots & \vdots \\ K_{2N-1,0} & K_{2N-1,1} & \dots & K_{2N-1,2N-1} \end{bmatrix}$$

FEM: Constraints

After r constrained DOFs removed:

$$K' \begin{bmatrix} q_0 \\ \vdots \\ q_{2N-r-1} \end{bmatrix} = \begin{bmatrix} F_0 \\ \vdots \\ F_{2N-r-1} \end{bmatrix}$$

- $(2N - r) \times (2N - r)$ matrix
- Non-degenerate: can solve for q

FEM: Displacements

Solve for q

1 DOF:

$$q = F / k$$

$(2N - r)$ DOFs:

$$\begin{bmatrix} q_0 \\ \vdots \\ q_{2N-r-1} \end{bmatrix} = (K')^{-1} \begin{bmatrix} F_0 \\ \vdots \\ F_{2N-r-1} \end{bmatrix}$$

FEM: Displacements: Gaussian elimination

Don't want to inverse K' ? Solve $K'q = F$ directly

Example: 2 unconstrained DOFs

$$K_{00}q_0 + K_{01}q_1 = F_0 \quad (1)$$

$$K_{10}q_0 + K_{11}q_1 = F_1 \quad (2)$$

How many (1)s do we subtract from (2) to eliminate $K_{10}q_0$? Answer: $\frac{K_{10}}{K_{00}}$

FEM: Displacements: Gaussian elimination

$$K_{00}q_0 + K_{01}q_1 = F_0$$

$$K_{10}q_0 + K_{11}q_1 = F_1$$



$$K_{00}q_0 + K_{01}q_1 = F_0$$

~~$$\left(K_{10} - \frac{K_{10}}{K_{00}}K_{00}\right)q_0 + \left(K_{11} - \frac{K_{10}}{K_{00}}K_{01}\right)q_1 = F_1 - \frac{K_{10}}{K_{00}}F_0$$~~

0



Find q_1 , then q_0

FEM: Stresses

Displacements interpolated linearly



Constant strain over element

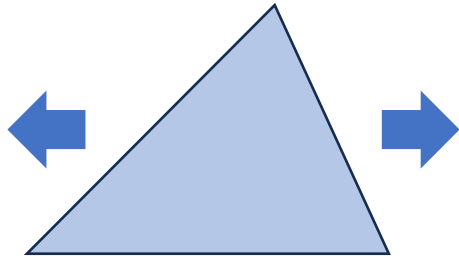


Constant stress over element

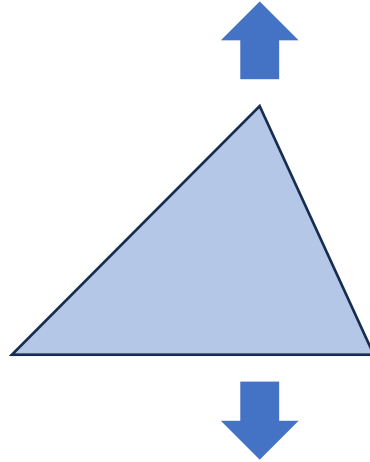


Constant color in stress visualization

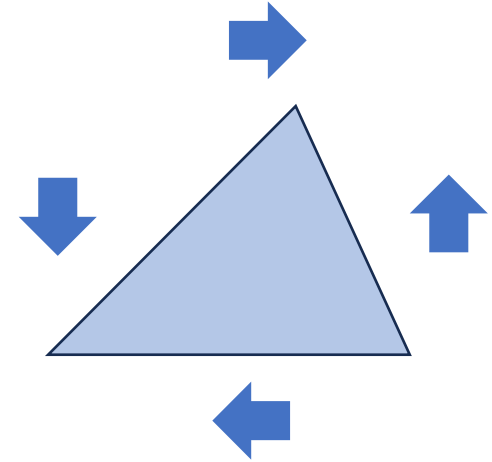
FEM: Stresses



X tension



Y tension

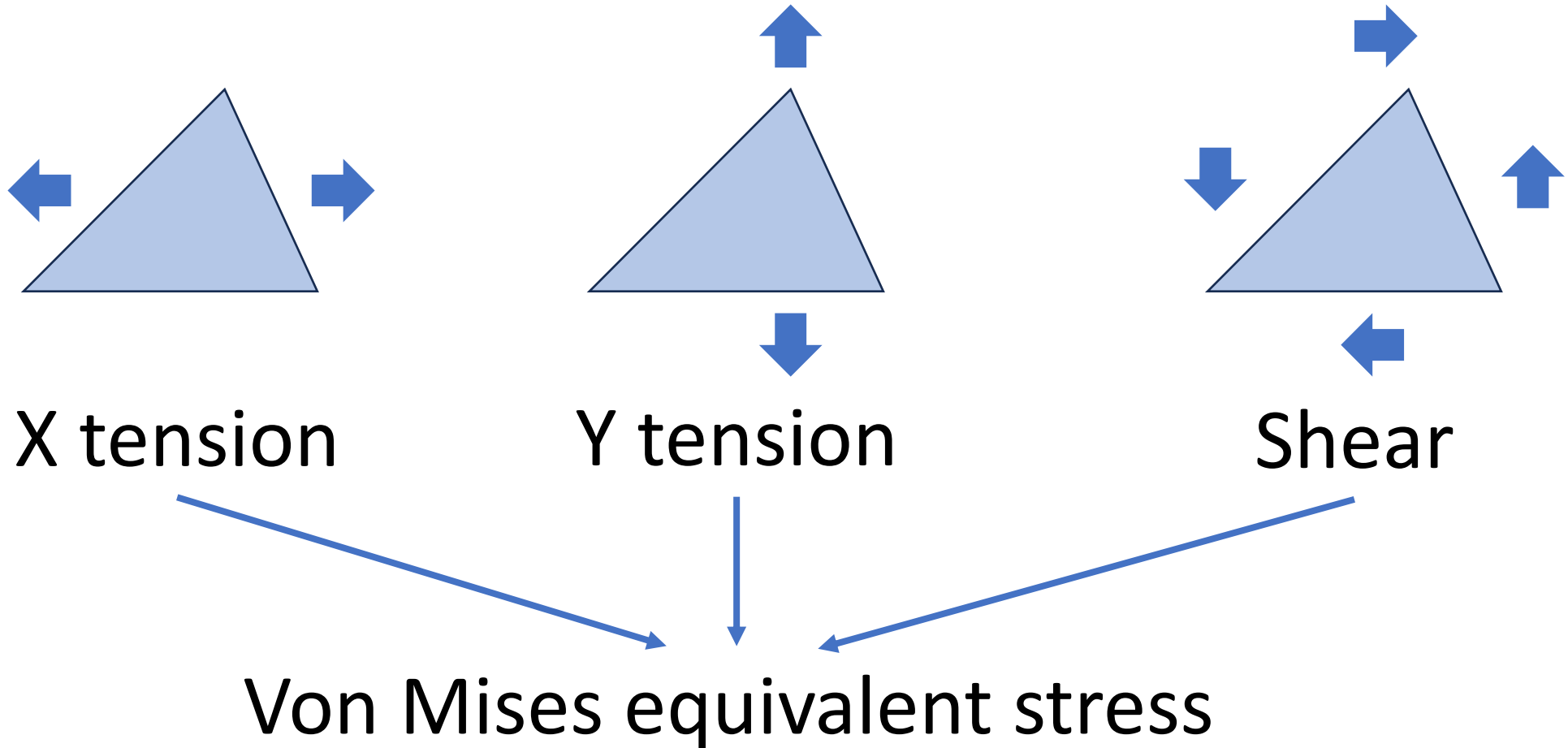


Shear

3 stresses responsible for 3 “elastic” DOFs

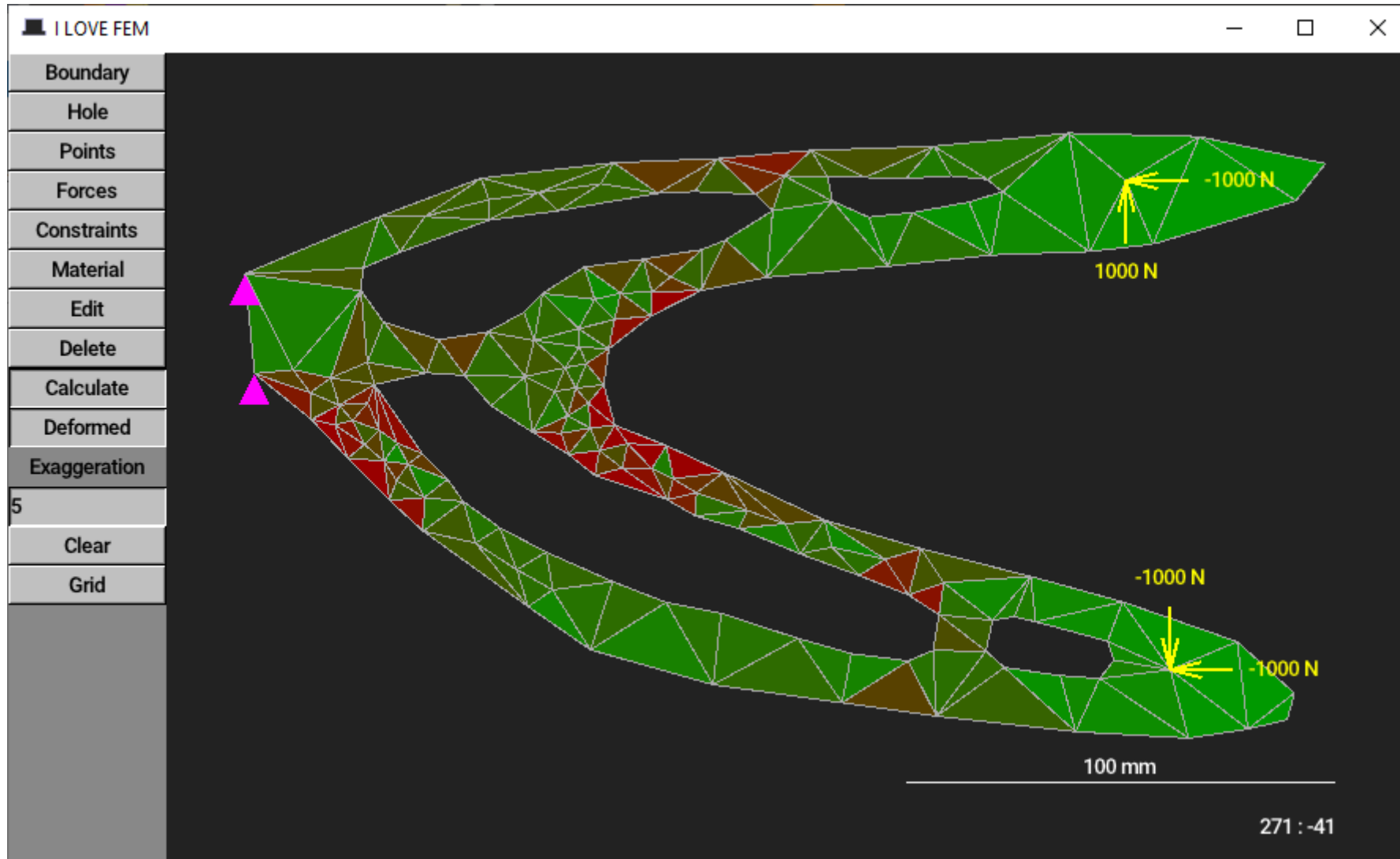
Explicit formula in `fem::getEquivalentStress()`

FEM: Stresses



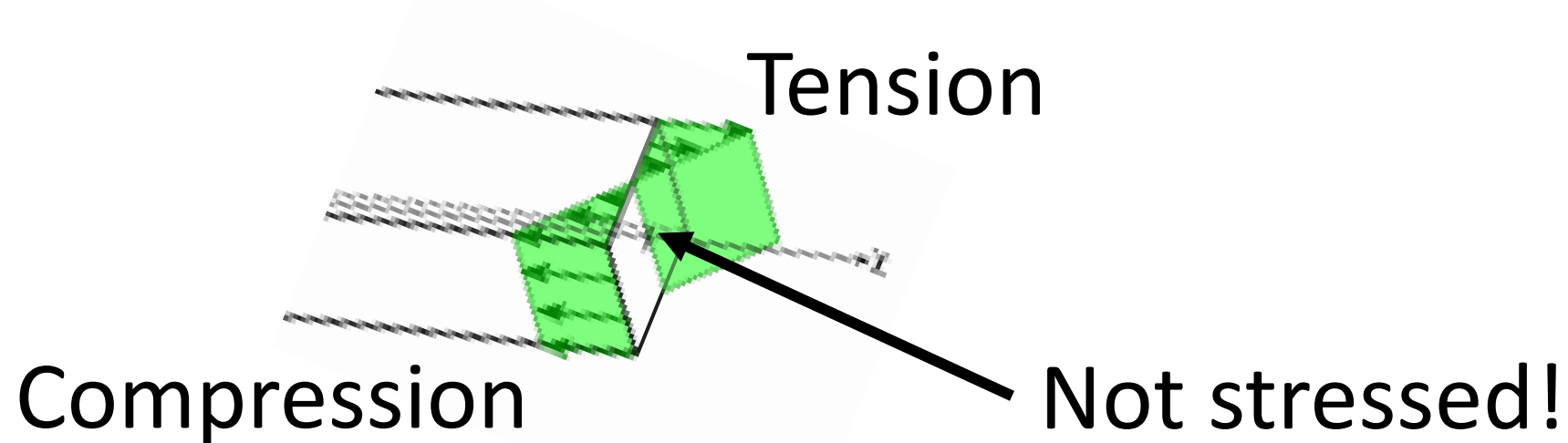
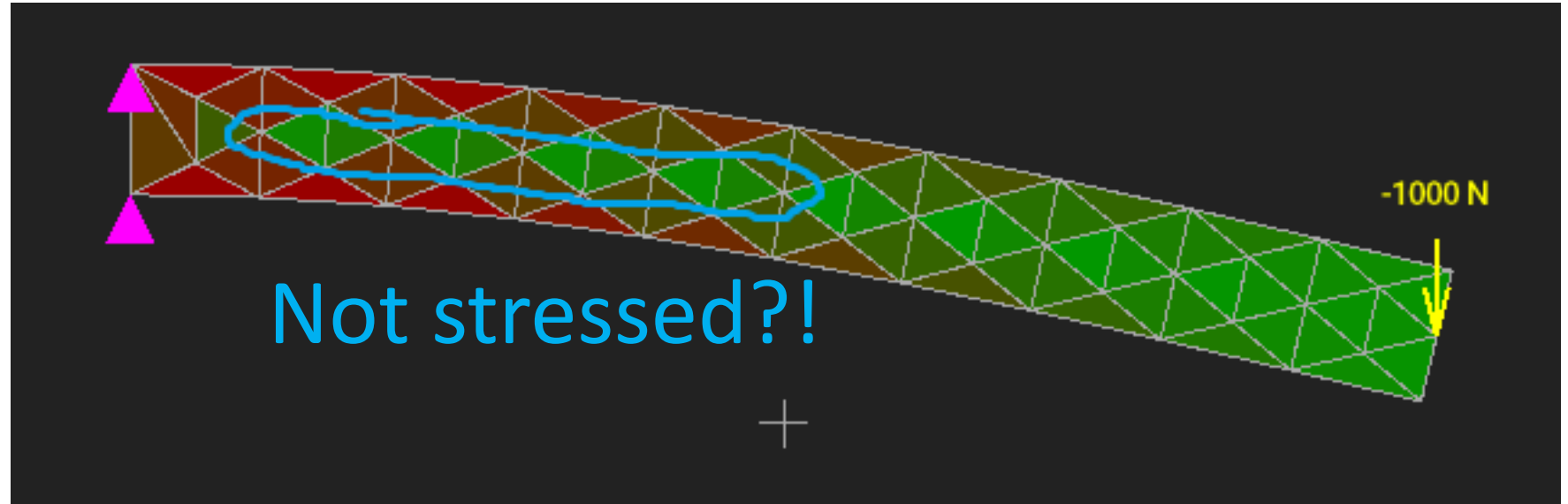
Explicit formula in `fem::getEquivalentStress()`

Visualization



Self-check

Beam
bending



Implementation



Umka: a statically typed
embeddable scripting language

<https://github.com/vtereshkov/umka-lang>



tophat: a 2D framework for
making games in Umka

<https://tophat2d.dev/>

Questions?