

# Contact of Rough Spheres: Greenwood-Tripp model

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## Model description

To make a truly predictive model linking contact area and force in contact of rough spheres, Greenwood and Tripp constructed a model taking into account averaged deformations induced by the contact pressure due to asperity contact [1]. This model, initially constructed for spherical contact, could be extended to arbitrary contact shape (its implementation for the axisymmetric contact is provided in [github.com/vyastreb/Greenwood-Tripp-Model](https://github.com/vyastreb/Greenwood-Tripp-Model)).

The main feature of the model is that it solves the problem in an iterative way by computing pressures from the separation field using multi-asperity model, then it computes induced displacements using integral convolution, recomputing separations and so on before the convergence. The pressure  $p^i(x, y)$  induced by asperity contact based on a statistical multi-asperity model for the separation  $z_0^i(x, y)$  between two surfaces is given by the following equation

$$p^i(x, y) = \frac{4}{3} \eta E^* r^{1/2} \int_{z_0^i(x, y)}^{\infty} (z - z_0^i(x, y))^{3/2} P(z) dz, \quad (1)$$

where  $\eta$  is the asperity density ( $m^{-2}$ ),  $r$  (m) their average geometrically mean curvature radius and  $P(z)$  ( $m^{-1}$ ) is the probability density of asperity heights. This average pressure is used to induce macroscopic deformation following Boussinesq approach, the total induced displacement (summing both sides) is given by

$$u^i(x, y) = \frac{1}{\pi E^*} \text{p.v.} \int_{\mathbb{R}^2} \frac{p^i(x', y') dx' dy'}{\sqrt{(x - x')^2 + (y - y')^2}}, \quad (2)$$

where p.v. denotes principal value of the integral. Then the separation between the surfaces is updated  $z_0^{i+1} = z_0^i + u^i$ , new pressure  $p^{i+1}$  is recomputed (1) as well as the induced displacement  $u^{i+1}$  from (2). For an axisymmetric problem, a simpler form is available [2][see Eq. (3.96a)]:

$$u_z(r) = \frac{4}{\pi E^*} \int_0^{\infty} \frac{\rho}{\rho + r} p(\rho) K(k(r, \rho)) d\rho \quad (3)$$

where  $r$  and  $\rho$  are the radial coordinate and the modulus  $k$  of the complete elliptic integral of the first kind  $K(k)$  is given by:

$$k(r, \rho) = \frac{4\rho r}{(r + \rho)^2}, \quad K(k) = \int_0^{\pi/2} [1 - k \sin^2(t)]^{-1/2} dt. \quad (4)$$

At the convergence separation  $z^*(x, y)$ , we have the information about contact area fraction, which can be computed as

$$A^*(x, y) = \pi \eta r \int_{z^*}^{\infty} (z^* - z) P(z) dz. \quad (5)$$

### Example

For the set of parameters provided in Table 1, the roughness parameter [3] can be evaluated

$$\alpha = \frac{\sigma R}{a^2} \approx 0.361,$$

placing this problem in the class of problems where the deviation from the Hertz theory is significant.

We use a relaxation technique by weighting the obtained displacement as

$$u'_{k+1} = \kappa u_{k+1} + (1 - \kappa) u_k,$$

Therefore, the change in displacement  $u'_{k+1} - u'_k = \kappa(u_{k+1} - u_k)$  and for the convergence we will require that

$$\frac{\|u_{k+1} - u_k\|_\infty}{\|u_k\|_\infty + \varepsilon_0} \leq \epsilon \Leftrightarrow \frac{\|u'_{k+1} - u_k\|_\infty}{\|u_k\|_\infty + \varepsilon_0} \leq \kappa \epsilon,$$

where  $\varepsilon_0$  is a small parameter, set by default to  $10^{-20}$ , and used to avoid division by zero. The iterative algorithm converges in 17 iterations for the tolerance of  $\epsilon = 10^{-3}$  defined by The obtained pressure, contact area distribution, deformed configuration and the resulting surface displacement are shown in Figure 1 and compared with the reference Hertz solution for the same load.

### References

1. J. A. Greenwood and J. H. Tripp (1967). *The elastic contact of rough spheres*. Journal of Applied Mechanics, **34**(1), 153–159. doi:[10.1115/1.3607616](https://doi.org/10.1115/1.3607616)
2. K. L. Johnson (1985). *Contact Mechanics*. Cambridge University Press, Cambridge. Ninth printing, 2003.
3. J. A. Greenwood, K. L. Johnson, and E. Matsubara (1984). *A surface roughness parameter in Hertz contact*. Wear, **100**(1–3), 47–57. doi:[10.1016/0043-1648\(84\)90005-X](https://doi.org/10.1016/0043-1648(84)90005-X)

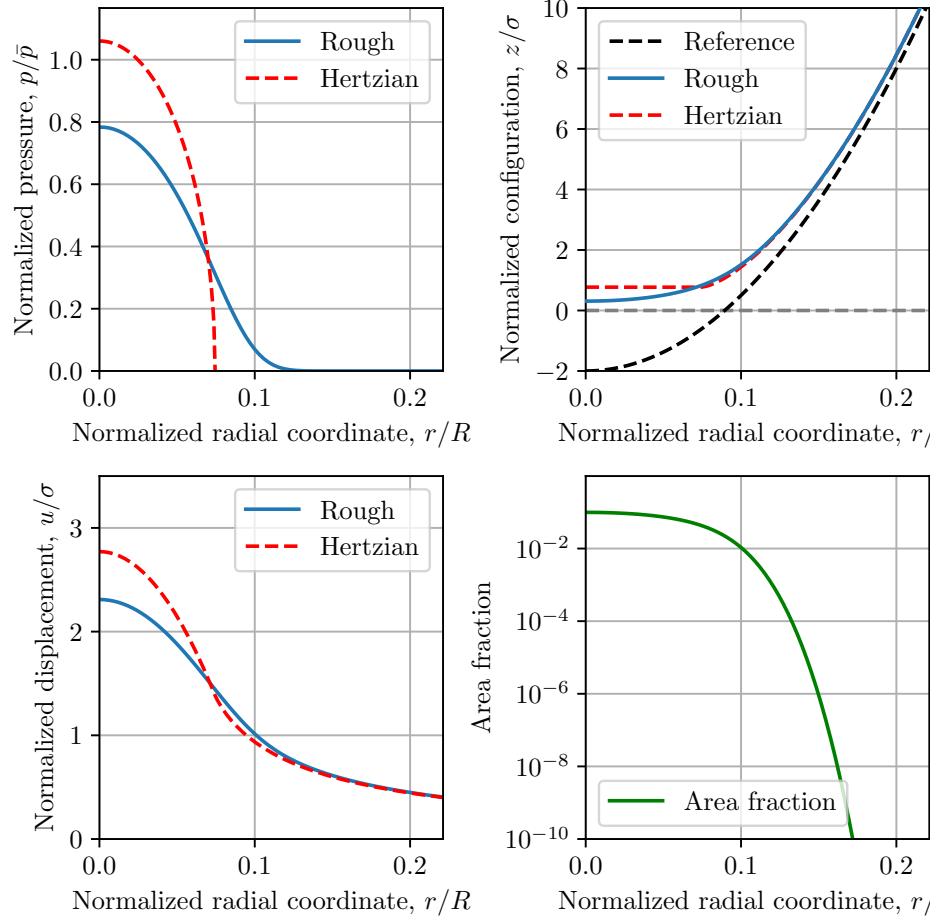


Figure 1: Converged pressure distribution and the corresponding Hertzian pressure (left column, upper pane); initial penetration (dashed line) and the resulting indenter's configuration for Greenwood-Tripp and Hertzian configuration (right column, upper panel); vertical displacement of Greenwood-Tripp and Hertzian displacement (left column, lower panel); contact area fraction for Greenwood-Tripp model (right column, lower panel).

Parameter	Symbol	Value	Units	Description
<b>Material Properties</b>				
Young's modulus	$E$	$2.1 \times 10^{11}$	Pa	Steel elastic modulus
Poisson's ratio	$\nu$	0.3	-	Poisson's ratio
Combined modulus	$E^*$	$\approx 1.15 \times 10^{11}$	Pa	$E/(2(1 - \nu^2))$
<b>Roughness Parameters</b>				
RMS roughness	$\sigma$	20	$\mu\text{m}$	Root mean square height
Asperity density	$\eta$	$2 \times 10^8$	$\text{m}^{-2}$	Number of asperities per unit area
Asperity tip radius	$\beta$	30	$\mu\text{m}$	Mean radius of curvature
<b>Indenter Geometry</b>				
Type	-	Sphere	-	Indenter shape
Radius	$R$	10	mm	Indenter radius
<b>Loading Parameters</b>				
Initial separation	$d$	-40	$\mu\text{m}$	$-2\sigma$
<b>Numerical Parameters</b>				
Convergence tolerance	$\epsilon$	$10^{-3}$	(-)	
Relaxation parameter	$\kappa$	0.2	(-)	

Table 1: Model parameters