



MA3005 Reference

Control Theory (Nanyang Technological University)



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- Final value theorem: $\lim_{t \rightarrow \infty} f(t) = \lim_{s \rightarrow 0} sF(s)$. Poles of $F(s)$ lie in left half s-plane and at most 1 pole on imaginary axis. [Steady state behaviour]
- Initial value theorem: $f(0^+) = \lim_{s \rightarrow \infty} sF(s)$. No restriction on poles location, applicable for sinusoidal functions.
- Linear System: Principle of superposition is applicable;

Time-Invariant System: If, $u(t) \rightarrow \boxed{G(s)} \rightarrow y(t)$ Then, $u(t-\tau) \rightarrow \boxed{G(s)} \rightarrow y(t-\tau)$

- Consider TF: $G(s) = \frac{(s+z_1)(s+z_2)}{s^2(s+p_1)(s+p_2)^3}$
 - $\angle s = -p_1$ is a simple pole when $n = 1$ (power)
 - $\angle s = 0$ is a pole of multiplicity 2
 - $\angle s = -p_2$ is a pole of multiplicity 3
 - $G(s) = 0$ at $-z_1$ and $-z_2$ (finite zeros)
 - there are 4 infinite zeros

- Open-Loop TF: $\frac{B(s)}{E(s)} = G(s)H(s)$ Forward Loop TF: $\frac{C(s)}{E(s)} = G(s)$

For negative (positive) feedback system, $CLTF = \frac{\text{Forward loop TF}}{1 \pm \text{Open loop TF}}$

For unity feedback system, $CLTF = \frac{\text{Forward loop TF}}{1 + \text{Forward loop TF}}$

- Notice directions of forces and displacements; assumptions relating displacements

- System error [Identify desired input and output]:

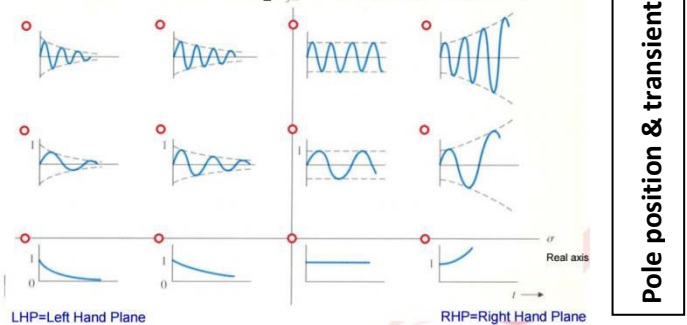
$E(s) = U(s) - C(s)$ $e_{ss}(t) = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} s \cdot E(s)$

$E_m(s) \neq E(s)$

8. Natural response:

- Stability, impulse input**
- Stable**
A system is called stable, if the output of the system converges to a particular value/state for a constant input.
 - Unstable**
When a system is unstable, the output of the system goes to infinity for finite input (or constant input).
 - Marginally Stable**
A marginally stable system will have continues oscillatory output for a constant input.

Impulse input



9. Routh-Hurwitz (Closed Loop CE):

C.E.: $a_0 s^n + a_1 s^{n-1} + a_2 s^{n-2} + \dots + a_n = 0$ $a_0, a_1, \dots, a_{n-1}, a_n > 0$ (or < 0)

s^n	a_0	a_2	a_4	...
s^{n-1}	a_1	a_3	a_5	...
s^{n-2}	b_1	b_2	b_3	...
s^{n-3}	c_1	c_2	c_3	...
\vdots	\vdots	\vdots	\vdots	\vdots
s^1	α	0		
s^0	β	0		

From Characteristic Eqn.

Need to be computed

$b_1 = \frac{a_1 \cdot a_2 - a_0 \cdot a_3}{a_1}$, $b_2 = \frac{a_1 \cdot a_4 - a_0 \cdot a_5}{a_1}$

$c_1 = \frac{b_1 \cdot a_3 - a_1 \cdot b_2}{b_1}$, $c_2 = \frac{b_1 \cdot a_5 - a_1 \cdot b_3}{b_1}$

Refer to the 1st column:

Special case

- If there is no changing of sign \Rightarrow closed loop system is stable
- How many times the sign changes in the 1st column represents the number of poles in RHP
- If 1st element in any row is zero, but others are non-zero, replace 0 with $\epsilon > 0$ and proceed as before
- If sign above zero (ϵ) is the same as below, it indicates that there are a pair of imaginary roots
- If sign above zero (ϵ) is opposite to that of below it, it indicates that there is a sign change \Rightarrow unstable system

Occurrence of entire row of zeros

i.e. $(-\sigma, +\sigma)$, or $(-j\omega, +j\omega)$ Solution: Form auxiliary polynomial from the row immediately above all zeros row

10. $\frac{C(s)}{R(s)} = \frac{K}{Ts+1} = \frac{K/T}{s+1/T}$

T is time constant

$G(s) = \frac{C(s)}{R(s)} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$

$0 < \zeta < 1$

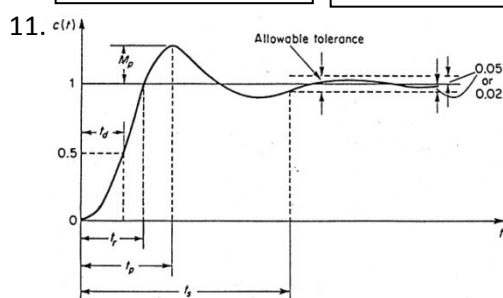
$s_{1,2} = -\zeta\omega_n \pm j\omega_n\sqrt{1-\zeta^2}$

$= -\sigma \pm j\omega_d$

$d^2(\text{ramp}) = d(\text{step}) = \text{impulse}$

Lower order approx.: Same gain + dominant pole

$\frac{1000}{(s+20)(s^2+6s+25)} \approx \frac{50}{s^2+6s+25}$



$t_r = \frac{1}{\omega_d} \tan^{-1} \left(\frac{\omega_d}{-\sigma} \right)$

$= \frac{\pi - \beta}{\omega_d} = \frac{\pi - \cos^{-1} \zeta}{\omega_d}$

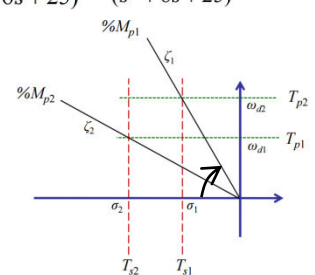
$t_p = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}} = \frac{\pi}{\omega_d}$

$t_s = 4T = \frac{4}{\sigma} = \frac{4}{\zeta\omega_n}$ (2% criterion)

$t_s = 3T = \frac{3}{\sigma} = \frac{3}{\zeta\omega_n}$ (5% criterion)

$\zeta = \cos \beta$

$\zeta = \frac{-\ln(M_p)}{\sqrt{\pi^2 + \ln^2(M_p)}}$



$\%M_p = e^{-\frac{\pi\zeta}{\sqrt{1-\zeta^2}}} \times 100$

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12.

Effects of increasing a parameter independently^[14]

Parameter	Rise time	Overshoot	Settling time	Steady-state error	Stability ^[11]
K_p	Decrease	Increase	Small change	Decrease	Degrade
K_i	Decrease	Increase	Increase	Eliminate	Degrade
K_d	Minor change	Decrease	Decrease	No effect in theory	Improve if K_d small

$$u(t) = K_p e(t) + \frac{K_p}{T_i} \int_0^t e(t) dt + K_p T_d \frac{de(t)}{dt}$$

$$\frac{U(s)}{E(s)} = K_p \left(1 + \frac{1}{T_i s} + T_d s \right) \quad \text{PI: eliminate steady state error.}$$

PD: anticipate error to **increase damping**, improve transient + stability.

13.

Unity-feedback system	Desired e_{ss}		Type 0		Type 1		Type 2	
	Input	Steady-state error formula	Static error constant	Error	Static error constant	Error	Static error constant	Error
	Step, $u(t) = 1$	$\frac{1}{1 + K_p}$	$K_p = \text{Constant}$	$\frac{1}{1 + K_p}$	$K_p = \infty$	0	$K_p = \infty$	0
	Ramp, $u(t) = t$	$\frac{1}{K_v}$	$K_v = 0$	∞	$K_v = \text{Constant}$	$\frac{1}{K_v}$	$K_v = \infty$	0
	Parabola, $u(t) = \frac{1}{2} t^2$	$\frac{1}{K_a}$	$K_a = 0$	∞	$K_a = 0$	∞	$K_a = \text{Constant}$	$\frac{1}{K_a}$

$$K_p = \lim_{s \rightarrow 0} G(s) = G(0)$$

$$K_v = \lim_{s \rightarrow 0} sG(s)$$

$$K_a = \lim_{s \rightarrow 0} s^2 G(s)$$

 $G(s)$

Open loop

transfer function

Fix type 0, 1 or 2

14. Sketching of root locus and its procedure: $KN(s) + D(s) = 0$

$$\text{Compute the poles (roots of } D) \text{ and the zeros (roots of } N)$$

$$n_{\text{poles}} - n_{\text{zeros}} \quad \sigma_a = \frac{\sum \text{poles} - \sum \text{zeros}}{n_{\text{poles}} - n_{\text{zeros}}}$$

For each complex pole (resp. zero), compute the departure (resp. arrival) angle by

$$\alpha_{\text{dep}}(p) = \pi - \sum_{\text{poles } p_i} \alpha_{p_i \rightarrow p} + \sum_{\text{zeros } z_i} \alpha_{z_i \rightarrow p} \quad \alpha_{\text{arr}}(z) = \pi + \sum_{\text{poles } p_i} \alpha_{p_i \rightarrow z} - \sum_{\text{zeros } z_i} \alpha_{z_i \rightarrow z}$$

break-in/break-out points $N'(s)D(s) - D'(s)N(s) = 0$
crossing of the imaginary axis $KN(j\omega) + D(j\omega) = 0$

"Every point of the real axis that is on the left of an odd number of poles/zeros belongs to the root locus."



$$\angle \frac{D(s_{\text{desired}})}{N(s_{\text{desired}})} = \angle(-K) = \pi \quad K = \frac{\|D(s_{\text{desired}})\|}{\|N(s_{\text{desired}})\|}$$

Validity of 2nd order approximation for a particular value of K (CE & CLTF):

- non-dominant poles are far to the left;
- zeros are far to the left;
- zeros that are not far to the left are cancelled by poles

15. [Reshaping root locus] PD controller & Lead Compensator (improve transient behaviour by pass through s_{desired}): $t_s, M_p \downarrow$

* \angle_c ✓ \angle_c	PD: $K_D \left(s + \frac{K_P}{K_D} \right) = K(s + z_c)$	$\angle(s_{\text{desired}} + z_c) = \angle D(s_{\text{desired}}) - \angle N(s_{\text{desired}}) - \pi$	$K = \frac{\ D(s_{\text{desired}})\ }{\ s_{\text{desired}} + z_c\ \cdot \ N(s_{\text{desired}})\ }$
	Lead: $K \frac{s + z_c}{s + p_c}$	$\angle(s_{\text{desired}} + z_c) - \angle(s_{\text{desired}} + p_c) = \angle D(s_{\text{desired}}) - \angle N(s_{\text{desired}}) - \pi$	$K = \frac{\ s_{\text{desired}} + p_c\ \cdot \ D(s_{\text{desired}})\ }{\ s_{\text{desired}} + z_c\ \cdot \ N(s_{\text{desired}})\ }$

PD controller = ideal lead compensator; PD controllers cannot be realized by passive circuits and are sensitive to noises; PD controller yield best steady-state behaviour. Decreasing z_c degrades performance of **lead compensators**. [e_{ss} increases (at same Q point), slower transient]

16. [Reshaping root locus] PI controller & Lag Compensator (reduce steady state error, yet no major change in root locus):

PI: $K_P + \frac{K_I}{s} = \frac{K(s + z_c)}{s}$	z_c is very small transient behavior unchanged	increases the type of a given system by 1	eliminates steady-state error
Lag: $K \frac{s + z_c}{s + p_c}$	z_c and p_c are very small close to zero $p_c \ll z_c$ pole/zero cancellation	steady-state error has been reduced by a factor $\frac{p_c}{z_c}$	K value kept the same before and after compensated, small tolerance on desired roots .

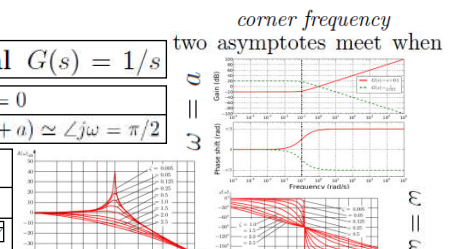
17. PID controller & Lead-Lag Compensator: forward transfer function $G(s)$

$$K_2 \frac{s + z_{c2}}{s} K_1 (s + z_{c1}) G(s) \quad K_2 \frac{s + z_{c2}}{s + p_{c2}} K_1 \frac{s + z_{c1}}{s + p_{c1}} G(s)$$

transients specifications \rightarrow
steady-state specifications \leftarrow

18. The steady-state response of a stable linear system with transfer function $G(s)$ to a sinusoidal input of the form $\sin(\omega t)$

$$c_{ss}(t) = \|G(j\omega)\| \sin(\omega t + \angle G(j\omega))$$

19. $20 \log(\|G(j\omega)\|)$ Y-axis $x = \log(\omega)$ $y = ax + b$ a straight lineDerivative $G(s) = s$ a straight line with slope +20 $\angle G(j\omega) = \angle j = \pi/2$ integral $G(s) = 1/s$ First-order terms $G(s) = s + a$ $\omega \rightarrow 0, 20 \log(\|j\omega + a\|) \simeq 20 \log(a)$ $\angle(j\omega + a) \simeq \angle a = 0$
 $\omega \rightarrow \infty, 20 \log(\|j\omega + a\|) \simeq 20 \log(\|j\omega\|) = 20 \log(\omega)$ $\angle(j\omega + a) \simeq \angle j\omega = \pi/2$ Second-order $G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$ $\omega \rightarrow 0, G(j\omega) \rightarrow 1$ $y = 0$
 $G(j\omega) = \frac{1}{1 - \frac{\omega^2}{\omega_n^2} + 2j\zeta\frac{\omega}{\omega_n}}$ $\omega \rightarrow \infty, G(j\omega) \simeq -\frac{\omega_n^2}{\omega^2}$ $20 \log(\|G(j\omega)\|) \simeq 40(\log(\omega_n) - \log(\omega))$ $y = -\pi$
 ζ has no influence on the asymptotes affects the transition around the corner frequency [0.707]

20. Bode plots of complex functions (Forward Loop TF, not closed loop):

asymptotes	Left part $\omega \simeq 0$ to $\omega = 0.1\omega_{\min}$	Middle part $\omega = 0.1\omega_{\min}$ to $\omega = 10\omega_{\max}$	Right part $\omega = 10\omega_{\max}$ to $\omega \simeq \infty$
	plots will be represented by their left asymptotes	sampling and numerical evaluations sample more densely around corner frequencies	plots will be represented by their right asymptotes
	system is of type k $G(s) = \frac{N(s)}{s^k D(s)}$ $20 \log(\ G(j\omega)\) \simeq 20 \log \left(\frac{ N(0) }{\omega^k D(0) } \right) = 20 \log \left(\frac{ N(0) }{ D(0) } \right) - 20k \log(\omega)$	$\angle G(j\omega) \simeq \angle(N(0)/D(0)) - k \angle(j\omega)$ $N(0)/D(0) > 0$ $y = -k\pi/2$ $N(0)/D(0) < 0$ $y = \pi - k\pi/2$ ϵ $A > 0$ $y = -n\pi/2$ δ $A < 0$ $y = \pi - n\pi/2$	system is of order n $G(s) = \frac{N(s)}{D(s)}$, $\deg(D) - \deg(N) = n$ $20 \log(\ G(j\omega)\) \simeq 20 \log \left(\frac{ A }{\omega^n} \right) = 20 \log A - 20n \log(\omega)$ A is the ratio of the leading coefficient of N to D

21. Bode plot analysis:

Left asymptote's gradient for system type, right asymptote's gradient for system order.	Type-0 $20 \log(\ G(0)\) = 20 \log K_p$ $K_p = 10^{\frac{y_{\text{asymptote}}}{20}}$ $e_{ss} = \frac{1}{1 + K_p}$	Refer No.13	Nyquist stability criterion
Type-1 $G(s) = \frac{N(s)}{sD(s)}$ $K_v = \frac{ N(0) }{ D(0) }$ $(x, y) = (0, 20 \log(N(0)/D(0)))$ $K_v = 10^{\frac{y_{\text{intersect}}}{20}}$ $e_{ss} = \frac{1}{K_v}$			TF is $G/(1 + G)$ Unity-Feedback System $\ G(j\omega)\ < 1$ when $\angle G(j\omega) = \pm \pi \pmod{2\pi}$ $\angle G(j\omega) > -\pi$ when $\ G(j\omega)\ = 1$ $G_M = -20 \log \ G(j\omega^*)\ $ $\angle G(j\omega^*) = -\pi$ $\phi_M = \pi + \angle G(j\omega^*)$ $\ G(j\omega^*)\ = 1$