Clarrical Mechanics (Phys 601) - November 8, 2011

* Euler's equations:

$$\overline{\Gamma}^{(e)} = \left(\frac{d\overline{L}}{dt}\right) = \left(\frac{d\overline{L}}{dt}\right) + \overline{\omega} \times \overline{L}$$

$$\left(\overline{L} = \overline{L} \overline{\omega}, \text{ with } \overline{L} \text{ constant in body frame}\right)$$

In principal axes frame:

or component-wise:

$$(\Gamma_1^{(e)} = I, \dot{\omega}_1 - \omega_2 \omega_3 (I_2 - I_3)$$

$$\left\langle \Gamma_{2}^{(e)} = T_{2}\dot{\omega}_{2} - \omega_{3}\omega_{1} \left(T_{3} - T_{1} \right) \right\rangle$$

$$\left[\Gamma_3^{(e)} = I_3 \dot{\omega}_3 - \omega_1 \omega_2 (I_1 - I_2)\right]$$

Note: if I is not constant

Ti(e) = d(Iiwi) + E Eijk w. wk Ik

There equations require the torque in the moving
rigid lody principal axes frame

Kinetic energy in principal axes frame:

$$T = \frac{1}{2} \omega^{T} I \omega = \frac{1}{2} (1, \omega,^{2} + T_{2} \omega_{2}^{2} + T_{3} \omega_{3}^{2})$$

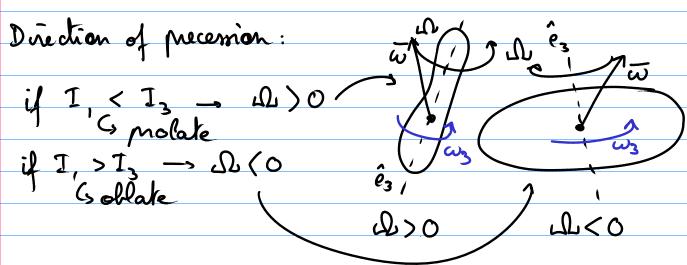
Equations of motion:

$$\begin{cases}
T, \dot{\omega}, &= \omega_{2} \omega_{3} & (T_{1} - T_{3}) \\
T_{2} \dot{\omega}_{1} &= \omega_{3} \omega_{3} & (T_{3} - T_{1}) \\
T_{3} \dot{\omega}_{3} &= \omega_{1} \omega_{2} & (T_{1} - T_{2}) & (T_{3} \dot{\omega}_{2} = 0) \Rightarrow \vec{\omega} \text{ constant} \\
T_{1}, \dot{\omega}_{1} &= \omega_{2} \omega_{3} & (T_{1} - T_{2}) & (T_{3} \dot{\omega}_{3} = 0)
\end{cases}$$

Symmetric top:

$$T_{1} = U_{1} \cup U_{2} \cup U_{3} \cup$$

puncipal exis êz



Magnitude of
$$\omega$$
:

 $\omega_1^2 + \omega_2^2 = \omega_1^2 = constant$
 $\omega^2 = \omega_1^2 + \omega_3^2 = constant magnitude of angular velocity$

At
$$t = t_0$$
, $\bar{\omega}$ in \hat{e} , \hat{e}_2 plane with angle λ between $\bar{\omega}$ and \hat{e}_3

$$= \left(\begin{array}{c} \omega_3(t_0) = \omega \cos \lambda \\ \omega_1(t_0) = \omega \sin \lambda \cos \Omega(t-t_0) \\ \omega_2(t_0) = \omega \sin \lambda \sin \Omega(t-t_0) \end{array} \right)$$

Recession of the earth's axis:

$$I_3-I_1\approx \frac{1}{305}$$
 (i.e., $I_3\approx I_1$) - modate - precession opposite to earth's λ small - $\omega_3\approx\omega$

$$\Rightarrow \Omega = \omega_3 \frac{I_3 - I_1}{I_1} \approx \frac{\omega}{305} \Rightarrow T = 305 \text{ days}$$

In reality: $T \approx 14$ months for $\lambda = 6 \times 10^{-7}$ rad = 4 m at poles

Angular momentum:

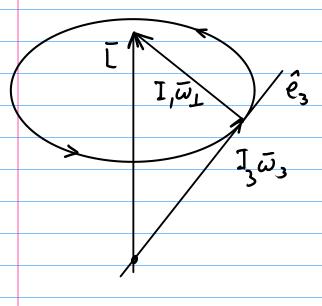
$$\Gamma^{(e)} = 0 \Rightarrow (\underline{dL}) = 0 \Rightarrow \overline{L} = constant of motion$$

and $\overline{L} = \overline{L} = \overline{$

$$|\bar{\omega}| = \sqrt{\omega_1^2 + \omega_3^2} = \text{constant of motion}$$

angle O between I in inertial frame and êz in body frame:

$$\cos \theta = \frac{\overline{L} \cdot \overline{u}_3}{|\overline{L}| \cdot |\overline{u}_3|}$$



Asymmetric top: $I_1 \neq I_2 \neq I_3 \rightarrow \text{general case}$

i)
$$T = \frac{1}{2}\omega^T I \omega = \frac{1}{2}(I, \omega_1^2 + I_2\omega_2^2 + I_3\omega_3^2) = E = combant$$
 of motion

2)
$$\Gamma^{(e)} = 0 \Rightarrow \bar{L} = constant in the inertial frameBecause magnitude $\bar{L} \cdot \bar{L}$ is independent of coordinate frame
 $\Rightarrow \bar{L} \cdot \bar{L} = constant of motion in any coordinate frame$$$

$$[= I \overline{\omega} = S \overline{L}^2 = \overline{I}_1^2 \omega_1^2 + \overline{I}_2^2 \omega_2^2 + \overline{I}_3^2 \omega_3^2 = \overline{L}^2 = constant of notion$$
Explicitly: $\frac{d}{dt}(\overline{L}^2) = 2 \overline{I}_1^2 \omega_1 \dot{\omega}_1 + 2 \overline{I}_3^2 \omega_2 \dot{\omega}_2 + 2 \overline{I}_3^2 \omega_3 \dot{\omega}_3$

=
$$2I_1 \omega_1 \omega_2 \omega_3 (I_2 - I_3) + 2I_2 \omega_2 \omega_3 \omega_1 (I_3 - I_1)$$

+ $2I_3 \omega_3 \omega_1 \omega_2 (I_1 - I_2)$
= 0

Carbitrary combants from initial conditions

$$\begin{cases} I_{1}\omega_{1}^{2} + I_{2}\omega_{2}^{2} = 2E - I_{3}\omega_{3}^{2} \\ I_{1}^{2}\omega_{1}^{2} + I_{2}^{2}\omega_{2}^{2} = L^{2} - I_{3}^{2}\omega_{3}^{2} \end{cases}$$

$$\left(\begin{array}{ccc} I_1 & I_2 \\ I_1^2 & I_2^2 \end{array}\right) \left(\begin{array}{c} \omega_1^2 \\ \omega_2^2 \end{array}\right) = \left(\begin{array}{ccc} 2E - I_3 \omega_3^2 \\ L^2 - I_3^2 \omega_3^2 \end{array}\right)$$

Now insert w, and wz: $I_3 \dot{\omega}_3 = \omega_1 \omega_2 (I_1 - I_2)$ \Rightarrow differential equation for $\omega_3(I)$ (hard to solve) Skalrlity of robitions around principal oxes Special case: consider $\bar{w} = \omega_i \hat{e}_i$ along principal avoir $\left[I_1 \dot{\omega}_1 \cdot \omega_1 \omega_3 \left(I_2 - I_3 \right) = 0 \right]$ => w is constant and remain along oxis

Sequilibrium =) $I_{q} \dot{\omega}_{q} = \omega_{q} \omega_{q} (I_{q} - I_{r}) = 0$ $I_{q} \dot{\omega}_{q} = \omega_{q} (I_{r} - I_{q}) = 0$ Consider again: [I, w, + I, w2 + I, w3 = 2 E $\int_{1}^{2} w_{1}^{2} + T_{2}^{2} w_{2}^{2} + T_{3}^{2} w_{3}^{2} = L^{2}$ L2 L2 L3 Assume I3>I, > shortest semiaxis along ê, , etc

For any intersection to exist we must have $2EI, < L^2 < 2EI_3$

If L2 just larger than 2EI, -> path around ê, pole If L2 just smaller than 2EI3 -, path around ê3 pole

If Le = 2EI, - path through er pole

2D equivalent:

relation in the vicinity of ê, and ê, is skalle, but around
ê êz is unstable

Small oscillations:

$$\vec{\omega} = \begin{pmatrix} \eta_{1} \\ \eta_{2} \\ \omega_{0} + \eta_{3} \end{pmatrix} = \begin{pmatrix} I_{1} \dot{\eta}_{1} = \omega_{2} \omega_{3} & (I_{2} - I_{3}) = \eta_{2} \omega_{0} & (I_{2} - I_{3}) \\ I_{2} \dot{\eta}_{2} = \omega_{3} \omega_{1} & (I_{3} - I_{1}) = \omega_{0} \eta_{1} & (I_{3} - I_{1}) \\ I_{3} \dot{\eta}_{3} = \omega_{1} \omega_{2} & (I_{1} - I_{2}) = O + O(\eta_{1}, \eta_{2}) \end{pmatrix}$$

$$\Rightarrow \left(\begin{array}{c} \dot{\eta}_{1} = \omega_{0} \frac{\overline{J}_{2} - \overline{I}_{3}}{\overline{I}_{1}} \eta_{2} \\ \dot{\eta}_{2} = \omega_{0} \frac{\overline{J}_{3} - \overline{I}_{1}}{\overline{I}_{2}} \eta_{1} \end{array} \right) \Rightarrow \ddot{\eta}_{1} = \omega_{0}^{2} \frac{\overline{I}_{2} - \overline{I}_{3}}{\overline{I}_{1}} \frac{\overline{J}_{3} - \overline{I}_{1}}{\overline{I}_{2}} \eta_{1}$$

$$\left(\begin{array}{c} \dot{\eta}_{3} = 0 \\ \dot{\eta}_{3} = 0 \end{array} \right) = \ddot{\eta}_{1} = -\Omega^{2} \eta_{2}$$

$$\left(\begin{array}{c} \dot{\eta}_{3} = -\Omega^{2} \eta_{2} \end{array} \right)$$

with
$$\Omega^2 = \omega_0^2 \frac{(I_3 - I_1)(I_3 - I_2)}{I_1 I_2}$$

If $(I_3 - I_1)(I_3 - I_2) > 0$:

$$\begin{cases} \eta_1 = A, \cos(\Omega I + \varphi) \\ \eta_2 = A, \cos(\Omega I + \varphi) \end{cases}$$
when $I_3 > I$, and $I_3 > I_2$
or $I_3 < I_1$, and $I_3 < I_2$

$$\Rightarrow largest on smallest moment$$

* Motion with torque: Compound pendulum Rigid lody rotating around a fixed wis ez (not principal) (not through center of mass:) reference frame with \hat{e} , through center of mans $\Rightarrow \hat{R} = l\hat{e}$, Degree of freedom: φ Torque: $\Gamma^{(e)} = \hat{R} \times M\bar{g}$ $\Rightarrow \Gamma^{(e)}_{3} = -Mglsin\varphi$ $\frac{d\overline{L}}{dt} = \overline{\Gamma}^{(e)} \Leftrightarrow \frac{d}{dt} (\overline{I}\overline{\omega}) = \overline{\Gamma}^{(e)}) \overline{\omega} = (0,0,\dot{\varphi})$ $= I_{33} \dot{\varphi} = -Mg l \sin \varphi$ $I_{33} = \int d^3 \bar{\tau} \, \rho(\bar{\tau}) \, \left(\chi_1^2 + \chi_2^2 \right) = \int d^3 \bar{\tau} \, \rho(\bar{\tau}) \, \chi_{\perp}^2$ $\Rightarrow \dot{\varphi} = -\frac{Mgl}{I_{33}} \sin \varphi \approx -\frac{Mgl}{I_{33}} \varphi \quad \text{for small } \varphi$ s harmonic oscillations with frequency $\Omega_{Q}^{2} = \frac{MgR}{I_{2}} \quad (anound point Q)$

Radius of gypation:
$$\overline{T}_{33} = Mk^2$$

$$\overline{I}_{33} = Mk^2$$

$$\Rightarrow \Omega_{Q}^{2} = \frac{g\ell}{\sqrt{k^{2}+\ell^{2}}} = \frac{g}{\ell'}$$

with
$$\ell' = \frac{\ell^2 + \overline{\ell}^2}{\ell} = \ell + \frac{\overline{k}^2}{\ell} = \text{equivalent length}$$

equivalent M moments k \hat{e}_3

Consider now the oxis of notation in the point P

$$\hat{\theta}_{2} = \frac{Mg(\ell'-\ell)}{M((\ell'-\ell)^{2}+\bar{b}^{2})}$$
 (around point P)

$$= \frac{g}{\ell'-\ell+\frac{\bar{k}^2}{\ell'-\ell}} = \frac{g}{\frac{\bar{k}^2}{\ell}+\ell} = \Omega_Q^2$$

* Rolling and Miding Prilliand ball

Ja ve stipping rolling
$$\hat{e}_3$$
 \hat{e}_2
 $I_1 = I_2 = I_3 = \frac{2}{5} Ma^2$

friction
$$F_{\parallel} = \mu F_{\perp} = -\mu g M \hat{e}_{\perp} \Rightarrow \tilde{z} = -\mu g$$
 (1)

$$\frac{dL_3}{dt} = J_3 \ddot{y} = F_{\parallel} \alpha \Rightarrow a \ddot{y} = \frac{5}{2} \mu g \qquad (3)$$

Hit center of ball:

— pure obiding immediately after impulse

(1)
$$\Rightarrow \dot{x} = v_0 - \mu g t$$

(2) $\Rightarrow a \dot{\varphi} = \frac{5}{2} \mu g t$

($\varphi(0) = 0, \dot{\varphi}(0) = 0$)

$$\dot{x} = \alpha \dot{\varphi} \iff v_0 - \mu g t = \frac{5}{2} \mu g r \iff t_1 = \frac{2}{7} \frac{v_0}{\mu g}$$

$$\Rightarrow \begin{cases} \dot{x}(t_1) = v_0 - \mu g t_1 = \frac{5}{7} v_0 \\ x(t_1) = v_0 t_1 - \frac{1}{7} \mu g t_1 = \frac{2}{7} \frac{v_0}{\mu g} - \frac{1}{7} \frac{4}{7} \frac{v_0^2}{\mu g} = \frac{12}{49} \frac{v_0^3}{\mu g}$$

Hit a height h above center:

$$\begin{array}{c}
\hat{e}_3 \\
\hat{e}_z
\end{array}$$

$$\Delta p_{\times} = \int F_{\lambda}(t) dt = impulse = Mv_{0}$$

$$\Rightarrow h M v_o = I \omega_o \Leftrightarrow h v_o = \frac{2}{5} a^2 \omega_o$$

1)
$$h = \frac{2}{5}a \Rightarrow v_0 = a\omega_0 \rightarrow \text{mo slipping because}$$

$$\dot{x} = a\dot{y} \quad \text{at } t = 0$$

2)
$$h \left(\frac{2}{5}a \Rightarrow v_0 > a \omega_0 \rightarrow \text{slipping and friction}\right)$$
relands the ball

(see above for $h=0$)