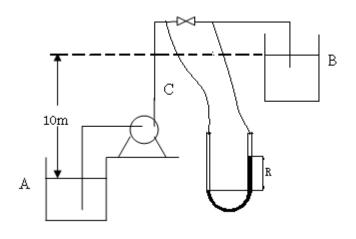
例 1. 用图示系统往高位槽输水,输水管径 φ 108×4mm,管路总长度为 100m(包括所有局部阻力的当量长度),摩擦系数 λ = 0.03,阀门的局部阻力系数为 9.5, U 形管压差计接在阀门两侧。泵特性方程: $H=30-0.0042~qv^2~(H-m;~qv-m^3/s)$ 。试求:(1) 管路中的流量;(2) 泵的有效功率;(3) 汞柱压差计读数 R。



解: (1) 在 1-1 截面与 2-2 截面间列伯努利方程

$$z_1 + \frac{p_1}{\rho g} + \frac{u_1^2}{2g} + H = z_2 + \frac{p_2}{\rho g} + \frac{u_2^2}{2g} + \sum H_f$$

 $z_1=0$, $p_1=p_2=0$ (表压), $z_2=10$ m , $u_1=u_2=0$,

$$\sum H_f = \lambda \frac{l + \sum l_e}{d} \frac{u^2}{2g} = 8\lambda \frac{l + \sum l_e}{g \pi^2 d^5} q_V^2 = \frac{8 \times 0.03 \times 100}{9.81 \times 3.14^2 \times 0.1^5} q_V^2 = 24813 q_V^2$$

故,管路特性方程: $H = z_2 + \sum H_f = 10 + 24813q_V^2$

联立泵特性方程与管路特性方程

$$H = 30 - 0.0042q_V^2$$

 $H = 10 + 24813q_V^2$ 得 $q_V = 0.0284$ m/s, $H = 30$ m

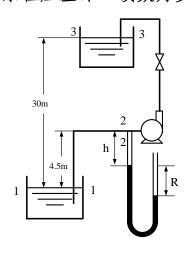
(2)
$$P_e = \rho g q_v H = 1000 \times 9.81 \times 0.0284 \times 30 = 8358W$$

(3)
$$u = \frac{q_V}{\frac{\pi}{4}d^2} = \frac{0.0284}{0.785 \times 0.1^2} = 3.62 \text{m/s}$$

由 B.E.
$$\frac{p_3}{\rho} = \frac{p_4}{\rho} + W_f$$
, $W_f = \zeta \frac{u^2}{2}$ 得 $\Delta p = \rho \zeta \frac{u^2}{2} = 1000 \times 9.5 \times \frac{3.62^2}{2} = 62245.9$ Pa

$$\Delta p = R(\rho_0 - \rho)g \Rightarrow R = \frac{\Delta p}{(\rho_0 - \rho)g} = \frac{62245.9}{(13600 - 1000) \times 9.81} = 0.5$$
m

例 2. 如图示一输水系统,管路直径为 $\Phi 80 \times 2 \, \text{mm}$, 当流量为 $36 \, \text{m}^3/\text{h}$ 时,吸入管路的总压头损失为 $0.6 \, \text{m}$, 排出管路的总压头损失为 $0.8 \, \text{m}$, 吸入管轴线到 U 型管左侧汞面的垂直距离 $h = 0.5 \, \text{m}$, 大气压力为 $100 \, \text{kPa}$ 。其他尺寸如图示,试计算:1. 泵的扬程 he; 2. 泵入口压力;3. 汞柱压差计 R 读数为多少?



解: (1) 在 1-1 截面与 3-3 截面之间列柏努利方程

$$z_1 + \frac{p_1}{\rho g} + \frac{u_1^2}{2g} + H_e = z_3 + \frac{p_3}{\rho g} + \frac{u_3^2}{2g} + H_{f1-3}$$

$$z_1 = 0m$$
, $z_3 = 30m$ $u_1 = 0m/s$, $u_3 = 0m/s$ $p_1 = 0kPa$ (表压), $p_3 = 0KPa$ (表

压)

$$H_{f1-3} = H_{f1-2} + H_{f2-3} = 0.6m + 0.8m = 1.4m$$

$$H_e = Z_3 + H_{f1-3} = 30 + 1.4 = 31.4m$$

(2) 在 1-1 截面与 2-2 截面之间列柏努利方程

$$z_1 + \frac{p_1}{\rho g} + \frac{u_1^2}{2g} = z_2 + \frac{p_2}{\rho g} + \frac{u_2^2}{2g} + H_{f1-2}$$

$$z_1 = 0m$$
, $z_2 = 4.5m$, $u_1 = 0m/s$, $u_2 = \frac{36}{3600 \times \frac{\pi}{4} \times 0.076^2} = 2.204m/s$

$$p_1 = 100 kPa$$
 (绝压) $H_{f1-2} = 0.6m$

$$p_2 = p_1 - \left(z_2 + \frac{u_2^2}{2g} + H_{f1-2}\right) \rho g$$

$$= 100 \times 10^{3} - \left(4.5 + \frac{2.204^{2}}{2 \times 9.81} + 0.6\right) \times 1000 \times 9.81 = 47540 Pa$$

(3) U 形管压差计列静力学方程: $p_2 + \rho g h + \rho_{Hg} g R = p_a$

$$R = (p_a - p_2 - \rho g h) / \rho_{Hg} g = 0.356 m$$