

Supplementary material

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I. SUPPLEMENTARY DESCRIPTIONS OF THE MODEL

IN this part the supplementary description about the derivation process of the state transition equation is given.

Based on reference [1], the state transition equation is derived as follows.

For a single pipeline, the dynamic changes in states, including the pressure π and mass flow rate q of gas flow, can be described by the following partial differential equations (PDEs):

$$\begin{aligned} \frac{\partial \pi}{\partial t} + \frac{c^2}{S} \frac{\partial q}{\partial x} &= 0 & 1 \\ \frac{\partial \pi}{\partial x} + \frac{1}{S} \frac{\partial q}{\partial t} + \frac{\lambda c^2}{2dS^2} \frac{q^2}{\pi} &= 0 & 2 \end{aligned}$$

Then, we introduce the average flow velocity method to process the nonlinear term $\frac{\lambda c^2}{2dS^2} \frac{q^2}{\pi}$ in the equation (2).

Due to the following relationships between gas pressure, mass flow rate, flow density and flow velocity:

$$\pi = c^2 \rho, \quad q = \rho v S \quad 3$$

So the nonlinear term in PDE (2) can be approximated as:

$$\frac{\lambda c^2}{2dS^2} \frac{q_i^2}{\pi_i} \approx \frac{\lambda c^2}{2dS^2} \frac{\rho_i^2 v_i^2 S}{c^2 \rho_i} = \frac{\lambda v_i^2}{2dS} q_i^t \quad 4$$

where v_i^t denotes the average flow velocity of the gas flow in pipe segment i during period t , which can be calculated by the following equation:

$$v_i^t = \frac{2}{3} \frac{v_{i+1}^{t+1}{}^3 - v_i^{t+1}{}^3}{v_{i+1}^{t+1}{}^2 - v_i^{t+1}{}^2} = \frac{2c^2}{3S} \frac{q_{i+1}^{t+1} \pi_i^{t+1}{}^2 + q_i^{t+1} \pi_{i+1}^{t+1} \pi_i^{t+1} + \pi_{i+1}^{t+1} q_i^{t+1}{}^2}{\pi_{i+1}^{t+1}{}^2 \pi_i^{t+1} + q_i^{t+1}{}^2 \pi_{i+1}^{t+1} q_{i+1}^{t+1}} \quad 5$$

By adopting the average flow velocity (4) and the Lax-Wendroff finite difference method, the PDEs (1) and (2) can be transformed into:

$$\begin{aligned} &\left(\frac{\pi_{i+1}^{t+1} - \pi_i^{t+1} + \pi_{i+1}^{t+1} - \pi_i^t}{\Delta t} \right) + \frac{c^2}{S} \left(\frac{q_{i+1}^{t+1} - q_i^{t+1} + q_{i+1}^t - q_i^t}{\Delta x} \right) = 0 & 6 \\ &\left(\frac{\pi_{i+1}^{t+1} - \pi_i^{t+1} + \pi_{i+1}^t - \pi_i^t}{\Delta x} \right) + \frac{1}{S} \left(\frac{q_{i+1}^{t+1} - q_i^{t+1} + q_{i+1}^t - q_i^t}{\Delta t} \right) + \\ &\quad \frac{\lambda \bar{v}_i^t}{4dS} (q_{i+1}^t + q_{i+1}^t + q_{i+1}^{t+1} + q_i^t) = 0 & (7) \end{aligned}$$

To facilitate the expression of the matrix elements, we define the following coefficients:

$$\begin{cases} c_1 = \frac{\pi^b}{\Delta t}, c_2 = \frac{c^2 q^b}{S \Delta x}, c_3 = \frac{\pi^b}{\Delta x}, \\ d_1 = q^b \left(\frac{1}{S \Delta t} + \frac{\lambda \bar{v}_i^t}{4dS} \right), d_2 = q^b \left(\frac{1}{S \Delta t} - \frac{\lambda \bar{v}_i^t}{4dS} \right) \end{cases} \quad 8$$

where π^b and q^b are the baseline values of the pressures and the mass flow rates, respectively.

Based on (1-8), the PDEs (1-6) and (1-7) of a discretized pipe segment i can be reformulated as follows:

$$\begin{bmatrix} c_1 & c_1 & -c_2 & c_2 \\ -c_3 & c_3 & d_1 & d_1 \end{bmatrix} \begin{bmatrix} \pi_i^{t+1} & \pi_{i+1}^{t+1} & q_i^{t+1} & q_{i+1}^{t+1} \end{bmatrix}^T = \begin{bmatrix} c_1 & c_1 & c_2 & -c_2 \\ c_3 & -c_3 & d_2 & d_2 \end{bmatrix} \begin{bmatrix} \pi_i^t & \pi_{i+1}^t & q_i^t & q_{i+1}^t \end{bmatrix}^T \quad 9$$

where the pressures and the mass flow rates are per-unit values.

Based on (9), the discretized PDE matrix of a single pipeline can be formulated as follows:

$$\begin{bmatrix} c_1 & c_1 & & & -c_2 & c_2 & & & \\ & c_1 & c_1 & & & -c_2 & c_2 & & \\ & & \ddots & & & & \ddots & & \\ & & & c_1 & c_1 & & & -c_2 & c_2 \\ -c_3 & c_3 & & & d_1 & d_1 & & & \\ & -c_3 & c_3 & & & d_1 & d_1 & & \\ & & \ddots & & & & \ddots & & \\ & & & -c_3 & c_3 & & & d_1 & d_1 \end{bmatrix} \begin{bmatrix} \pi_l^{t+1} \\ q_l^{t+1} \end{bmatrix} = \begin{bmatrix} c_1 & c_1 & & & c_2 & -c_2 & & & \\ & c_1 & c_1 & & & c_2 & -c_2 & & \\ & & \ddots & & & & \ddots & & \\ & & & c_1 & c_1 & & & c_2 & -c_2 \\ c_3 & -c_3 & & & d_2 & d_2 & & & \\ & c_3 & -c_3 & & & d_2 & d_2 & & \\ & & \ddots & & & & \ddots & & \\ & & & c_3 & -c_3 & & & d_2 & d_2 \end{bmatrix} \begin{bmatrix} \pi_l^t \\ q_l^t \end{bmatrix} \quad (10)$$

$$C_l^{t+1} \begin{bmatrix} \pi_l^{t+1} \\ q_l^{t+1} \end{bmatrix}^T = C_l^t \begin{bmatrix} \pi_l^t \\ q_l^t \end{bmatrix}^T \quad (11)$$

where $y_l^t = [\pi_l^t \quad q_l^t]^T$ denotes the vector of the pressures and mass flow rates in pipeline l at time t .

Equation (11) is a simplification of (10). Based on (11), the discretized PDE matrix for all pipelines in the natural gas system can be formulated as follows:

$$\begin{bmatrix} C_1^{t+1} & & \\ & C_2^{t+1} & \\ & & \ddots \\ & & & C_l^{t+1} \end{bmatrix} y_{t+1} = \begin{bmatrix} C_1^t & & \\ & C_2^t & \\ & & \ddots \\ & & & C_l^t \end{bmatrix} y_t \quad 12$$

$$C_{t+1}^{PN} y_{t+1} = C_t^{PN} y_t \quad 13$$

Equation (13) is a simplification of (12).

And there are the pressure continuity constraints, the flow balance constraints and compressor boost constraints at the junctions:

$$\pi_i^t = \pi_n^t, \forall i \in \Omega_n \quad 14$$

$$\sum_{i \in \Omega_n^I} q_i^t - \sum_{j \in \Omega_n^O} q_j^t - q_n^t = 0 \quad 15$$

$$\pi_j^t = \pi_i^t + \pi_k^{CP}, \forall i \in \Omega_k^I, j \in \Omega_k^O \quad (16)$$

where π_n^t denotes the pressure of practical node n at time t ; Ω_n denotes the set of pressures π_i^t at the end of the pipeline connected to practical node n ; q_n^t denotes the gas load of practical node n , which includes the traditional gas load $q_{n,t}^{GL}$ and the gas-fired unit gas withdrawal $q_{n,t}^{GF}$; Ω_n^I and Ω_n^O denote the sets of gas flows into and out of practical node n , respectively; π_k^{CP} denotes boosted pressure provided by compressor k ; Ω_k^I and Ω_k^O denote the sets of gas flows into and out of compressor k .

Combined with these constraints, the dynamic state equation of the natural gas network can be formulated as follows:

$$C_t^{NGS} [\pi_t^{GW} \quad y_t \quad q_t^{GF} \quad q_t^{GL} \quad \pi_t^{CP}]^T = C_{t-1}^{NGS} [\pi_{t-1}^{GW} \quad y_{t-1}]^T \quad 17$$

where C_t^{NGS} denotes the coefficient matrix of the dynamic state equation of the natural gas system. Since different natural gas systems have different pressure continuity constraints, the flow balance constraints and compressor boost constraints, the parameters in C_t^{NGS} must be determined according to the specific topology of the natural gas system.

And with simple matrix operations, equation (17) can be transformed to:

$$O \lambda y_t = S \lambda y_{t-1} + I u_t \quad 18$$

where $O(\lambda)$ and $S(\lambda)$ denotes the coefficient matrices containing the pipeline friction factors λ to be identified; I is a constant matrix; $u_t = [\pi_{t-1}^{GW} \quad \pi_t^{GW} \quad \pi_t^{CP} \quad q_t^{GL} \quad q_t^{GF}]^T$ denotes the control variables at time t .

Then through the elementary row transformation of the matrix, y_t can be decomposed into the state variables y_t^{obs} of the observable nodes and the state variables y_t^{nob} of the unobservable nodes, and the equation (18) can be transformed to:

$$\begin{bmatrix} y_t^{obs} \\ y_t^{nob} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} \begin{bmatrix} y_{t-1}^{obs} \\ y_{t-1}^{nob} \end{bmatrix} + \begin{bmatrix} B_1 \\ B_2 \end{bmatrix} u_t \quad 19$$

Then through the recursive relationship and equation (19), we can get:

$$\begin{cases} \begin{bmatrix} y_t^{obs} \\ y_t^{nob} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} \begin{bmatrix} y_{t-1}^{obs} \\ y_{t-1}^{nob} \end{bmatrix} + \begin{bmatrix} B_1 \\ B_2 \end{bmatrix} u_t \\ \begin{bmatrix} y_{t-1}^{obs} \\ y_{t-1}^{nob} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} \begin{bmatrix} y_{t-2}^{obs} \\ y_{t-2}^{nob} \end{bmatrix} + \begin{bmatrix} B_1 \\ B_2 \end{bmatrix} u_{t-1} \\ \vdots \\ \begin{bmatrix} y_1^{obs} \\ y_1^{nob} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix} \begin{bmatrix} y_0^{obs} \\ y_0^{nob} \end{bmatrix} + \begin{bmatrix} B_1 \\ B_2 \end{bmatrix} u_1 \end{cases} \quad 20$$

After simplification, equation (20) can be transformed to:

$$\begin{bmatrix} y_t^{obs} \\ y_t^{nob} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}^t \begin{bmatrix} y_0^{obs} \\ y_0^{nob} \end{bmatrix} + \sum_{\tau=0}^{t-1} \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}^{\tau} \begin{bmatrix} B_1 \\ B_2 \end{bmatrix} u_{t-\tau} \quad 21$$

Then let $C_t = \begin{bmatrix} C_{1,t} & C_{2,t} \\ C_{3,t} & C_{4,t} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}^t$ and $D_t = \begin{bmatrix} D_{1,t} \\ D_{2,t} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}^t \begin{bmatrix} B_1 \\ B_2 \end{bmatrix}$, equation (21) can be transformed to:

$$\begin{bmatrix} y_t^{obs} \\ y_t^{nob} \end{bmatrix} = \begin{bmatrix} C_{1,t} & C_{2,t} \\ C_{3,t} & C_{4,t} \end{bmatrix} \begin{bmatrix} y_0^{obs} \\ y_0^{nob} \end{bmatrix} + \sum_{\tau=0}^{t-1} \begin{bmatrix} D_{1,\tau} \\ D_{2,\tau} \end{bmatrix} u_{t-\tau} \quad 22$$

It is clear that from the equation (22) we can get:

$$y_t^{obs} = C_{1,t} y_0^{obs} + C_{2,t} y_0^{nob} + \sum_{\tau=0}^{t-1} D_{1,\tau} u_{t-\tau} \quad 23$$

Matrices C and D are mathematically related to matrices A and B as follows:

$$\begin{bmatrix} C_{1,t} & C_{2,t} \\ C_{3,t} & C_{4,t} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}^t, \quad \begin{bmatrix} D_{1,t} \\ D_{2,t} \end{bmatrix} = \begin{bmatrix} A_{11} & A_{12} \\ A_{21} & A_{22} \end{bmatrix}^t \begin{bmatrix} B_1 \\ B_2 \end{bmatrix} \quad 24$$

From this, we obtain two forms of state transition equations (19) and (23).

REFERENCES

- [1] P. Zhao, Z. Li, X. Bai, J. Su, and X. Chang, "Stochastic real-time dispatch considering AGC and electric-gas dynamic interaction: Fine-grained modeling and noniterative decentralized solutions," *Applied Energy*, vol. 375, p. 123976, Dec. 2024.