

OPEN LOOP DYNAMICS

$$I\ddot{\theta} + f(\theta, \dot{\theta}) = \tau$$

NONLINEAR GRAVITY/FRICTION

\hat{f} = MODEL OF f 'S REAL EFFECT (WE'D MODEL FRICTION)

CONTROL LAW: $\tau = K_p \tilde{\theta} + K_d \dot{\tilde{\theta}} + \hat{f}(\theta, \dot{\theta})$

MODEL OF GRAVITY/FRICTION

WHAT IS $\|\hat{f}(\theta, \dot{\theta}) - f(\theta, \dot{\theta})\| = \|\tilde{f}(\theta, \dot{\theta})\|$

MODEL ERROR

WE CAN'T GUARANTEE ASYMPTOTIC STABILITY USING PD CONTROL $\tilde{\theta} = 0$

IF WE WANT $\tilde{\theta} \rightarrow 0$, USE PID CONTROL

CONTROL LAW: $\tau = K_p \tilde{\theta} + K_d \dot{\tilde{\theta}} + K_i \xi + \hat{f}(\theta, \dot{\theta})$

$\dot{\xi} = \tilde{\theta} \therefore \xi = \int \tilde{\theta}$

LYAPUNOV CANDIDATE

$V(\tilde{\theta}, \dot{\tilde{\theta}}, \xi)$

NEW 3RD STATE
(3RD ORDER SYSTEM)
• WE WANT ALL ERROR TO GO TO ZERO

LYAPUNOV RESULTS

• GIVEN CERTAIN CONDITIONS + BOUNDS ON THE GAINS, WE CAN SHOW THE SYSTEM IS ASYMPTOTICALLY STABLE BY LYAPUNOV/LASALLE

MORE GENERALLY, WE CAN SHOW PID CONTROL IS ASYMPTOTICALLY STABLE FOR A MULTI-DOF ROBOT

PROOF DONE BY:

[ARIMOTO + MIYAZAKI, 1983]

PROOF TO GUARANTEE PID CONTROL ON A MULTI-DOF ROBOT
(IT'S A MESS, IT'S DONE :))

GUARANTEED ERROR GOING TO ZERO...
GIVEN INFINITE TIME

NO GUARANTEE HOW FAST $\tilde{\theta} \rightarrow 0$

CAN WE COMPENSATE FOR ALL NONLINEARITIES IN A WAY THAT IMPROVES TRACKING PERFORMANCE?

SPLIT OPEN-LOOP DYNAMICS INTO LINEAR + NON-LINEAR DYNAMICS

$$\tau = [\underbrace{I + H^*(\theta)}_{\text{NON-LINEAR}}] \ddot{\theta} + \underbrace{V(\theta, \dot{\theta}) + B\dot{\theta}}_{\text{LINEAR DAMPING}} + \underbrace{F^*(\dot{\theta})}_{\text{NON-LINEAR DAMPING}} + G(\theta)$$

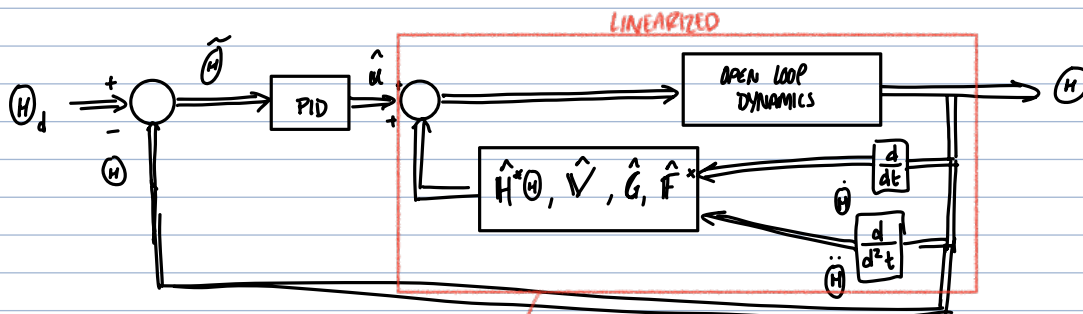
$$\text{CONTROL LAW: } \tau = \underbrace{\left(K_p + K_d s + \frac{K_I}{s} \right)}_{\text{MATRICES}} \underbrace{\tilde{\theta}}_{\text{JOINT ERRORS}} + \underbrace{\hat{H}(\theta) \ddot{\theta} + \tilde{V}(\theta, \dot{\theta}) + G(\theta)}_{\text{MODEL OF NON-LINEAR DAMPING}} + \underbrace{F^*(\dot{\theta})}_{\text{MODEL OF NON-LINEAR DAMPING}}$$

IF WE WANT C.L.

① = ② IF WE HAVE PERFECT MODELS, THEY'LL CANCEL W/ NON-LINEAR TERMS

$$\therefore I \ddot{\theta} + B \dot{\theta} = \left(K_p + K_d s + \frac{K_I}{s} \right) \tilde{\theta}$$

RESULTING CLOSED LOOP DYNAMICS ARE LINEAR

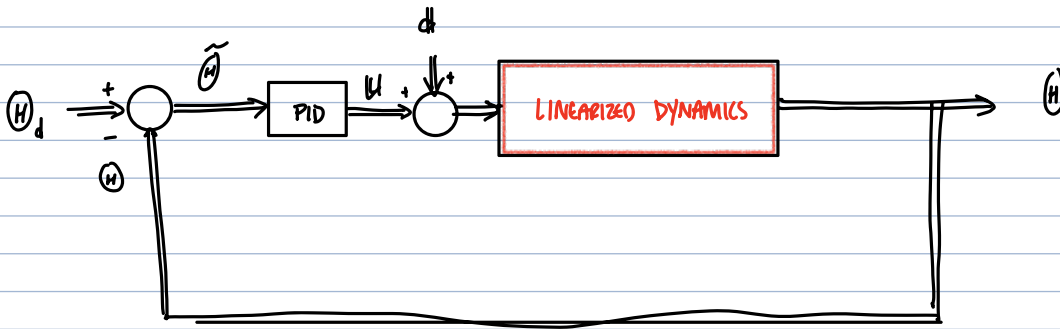


NOW PID CONTROLLER THINKS EVERYTHING IS LINEAR

DYNAMICS LINEARIZED WRT PID CONTROLLER

IF WE HAVE AN IMPERFECT MODEL, WE CAN VIEW LEFTOVER NON-LINEARITIES AS A DISTURBANCE ON OUR SYSTEM

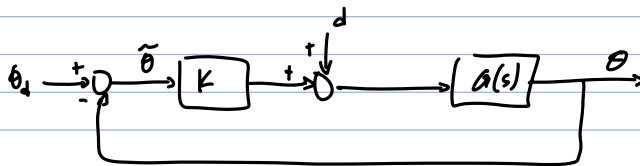
$$I \ddot{\theta} + B \dot{\theta} = u + d(\theta, \dot{\theta}, \ddot{\theta})$$



NOW WE WANT TO INCREASE GAINS TO BETTER MINIMIZE EFFECTS OF DISTURBANCE

⇒ LARGER PID GAINS ☺

eg. 1-DOF EXAMPLE



$$\theta(s) = G(s) \left[d(s) + \frac{K \tilde{\theta}(s)}{G(s)} \right]$$

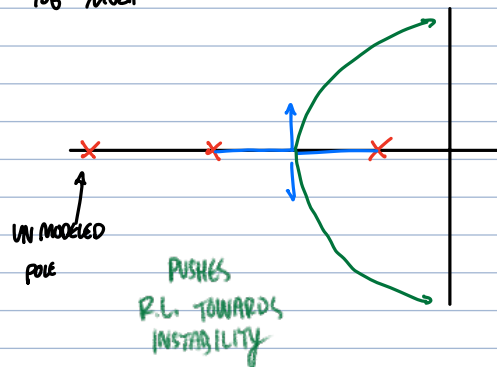
if $\tilde{\theta}_d = \theta$

$$\theta(s) = \left[\frac{G(s)}{1 + KG(s)} \right] d(s)$$

As $K \uparrow$, SMALLER EFFECT OF DISTURBANCE ON A TRAJECTORY

BUT IF WE INCREASE K TOO MUCH

(1) UNMODELLED POLES CREATE INSTABILITY



(2) CAN SATURATE AMPLIFIER



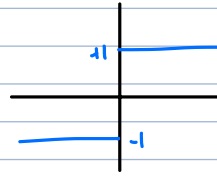
SATURATION IS A NON-LINEARITY THAT CREATES INSTABILITY (LIMIT CYCLING)

OTHER PROBLEMS:

- DIFFERENTIATING $\hat{\theta}$ TO GET $\dot{\hat{\theta}}$ & $\ddot{\hat{\theta}}$ AMPLIFIES SENSOR NOISE
- MEASURING $\dot{\theta}$ & $\ddot{\theta}$ IS ALSO NOISY
- NONLINEARITIES CAN AMPLIFY NOISE EVEN FURTHER

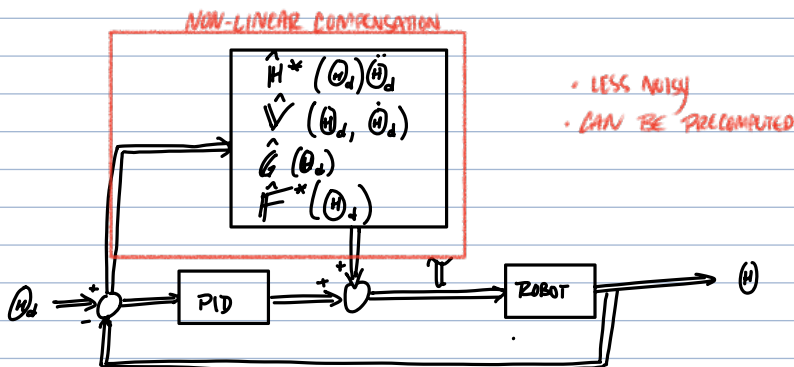
E.G. $\hat{F}(\dot{\theta}) = c \operatorname{sgn}(\dot{\theta})$

IF $\dot{\theta}$ HOVERS AROUND 0,
THEN \hat{F} WILL BOUNCE BACK
& FORTH (CHATTER)



ALTERNATIVE: FEEDFORWARD COMPUTED TORQUE CONTROL

- USE DESIRED TRAJECTORY RATHER THAN ACTUAL TRAJECTORY TO COMPUTE NON-LINEAR PARTS OF SYSTEM



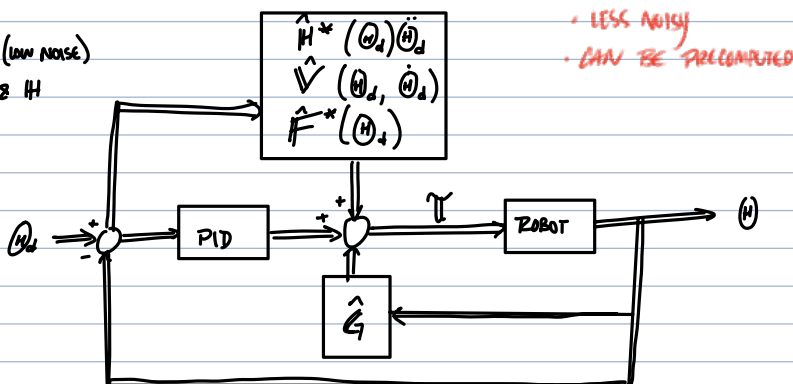
IF θ_d IS KNOWN AHEAD OF TIME, WE CAN PRE-COMPUTE ALL NON-LINEAR COMPENSATION

IF A DISTURBANCE KNOCKS YOU OFF, OR MODEL IS NOT VERY GOOD, THEN θ_d DEVIATES FROM θ , MAKING COMPENSATION INACCURATE

ALTERNATIVE: MIX OF FEED FORWARD & FEEDBACK COMPENSATION

EXAMPLE:

- FEEDBACK GRAVITY (LOW NOISE)
- FEED FORWARD IF & HI



THIS ALL MIGHT BE UNNECESSARY

• MAYBE $\dot{\theta}$ IS SMALL SO WE CAN LEAVE OUT \hat{V} COMPENSATION

• MAYBE ROBOT IS LIGHTWEIGHT OR ACCELERATING SLOWLY \hat{H} COMPENSATION OUT

IF $\ddot{\theta} \ll 1$ LEAVE \hat{H} OUT

IF $\dot{\theta} \ll 1$ LEAVE \hat{V} OUT

IF $N \gg 1$ $\therefore J_m$ DOMINATES \Rightarrow NEGLECT J_{arm} , NEGLECT \hat{H} COMPENSATION
 \uparrow
GEAR RATIO

REGARDLESS OF FEEDBACK VS FEEDFORWARD, PID CONTROL IS STILL DECENTRALIZED

ALL OF THESE CONTROLLERS ARE DECENTRALIZED

CONTROL EFFORT u_i (AND THUS τ_i)
DEPEND ONLY ON $\hat{\theta}_i$

\uparrow
OUR DYNAMICS ARE COUPLED, THIS IS AN ISSUE

WE NEED A CENTRALIZED CONTROLLER THAT LINEARIZES AND DECOUPLES THE PID CONTROL ACTION