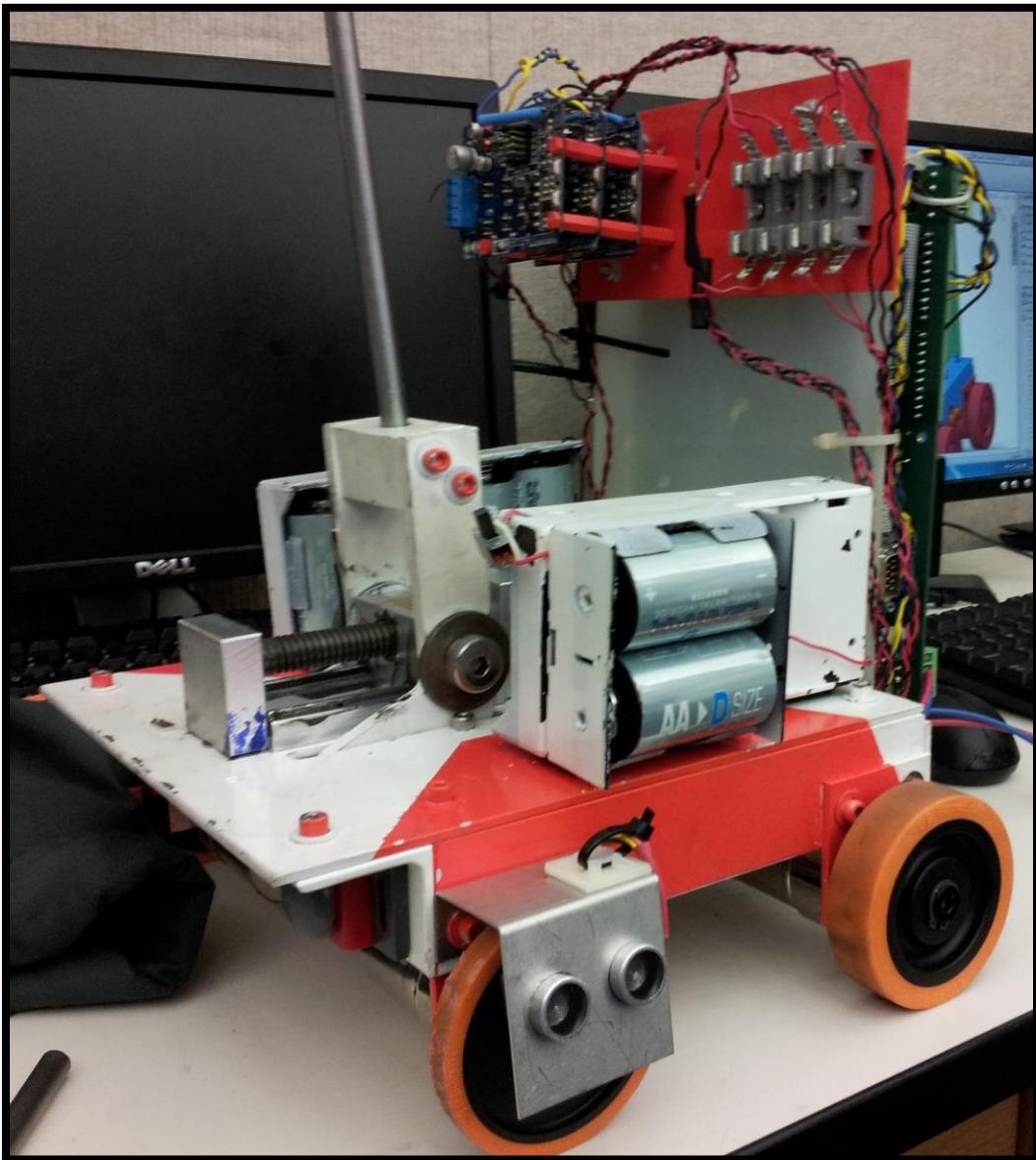


Motor Sizing Calculations Report

Panthra Bulk Material Transporter



Team 3
Alex Noe
Chris Underhill
Asher Katz
Wilson Lam
David Walter

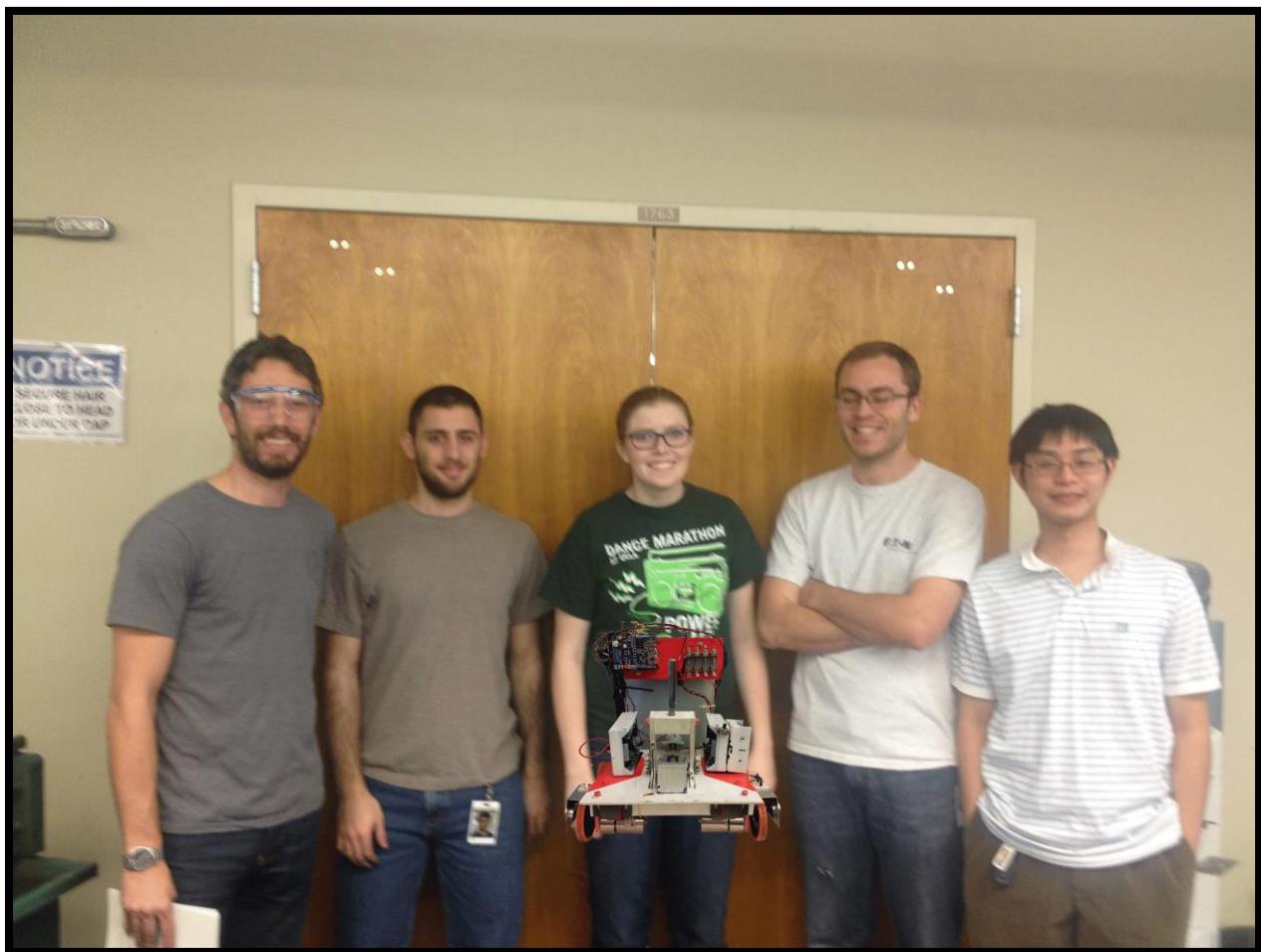


Figure 1: Team 3 (from left to right: David Walter, Asher Katz, Alex Noe, Chris Underhill, and Wilson Lam)

Abstract

The following final report outlines the design of our transporter *Panthra*. We start by outlining the design requirements for our transporter. Then we move on to our first three original designs and analyze the strengths and weaknesses of each, showing why we selected our *Panthra* design. We then go into a more detailed design description about the five subsystems of our transporter: the vehicle delivery system, the payload delivery system, the vehicle support structure, the electronics and control system, and the vehicle sensor array. Next we analyze power requirements for our transporter and the motor sizing requirements. These are calculations that we performed in our first three reports but with the most up to date numbers. We then go into the fabrication and testing of our transporter. . Lastly, we compare our design requirements with our design analysis to conclude that our design is satisfactory.

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List of Symbols

Symbols	Descriptions	Units
F_m	Gravitational force on the machine	lb_f
M	Mass of the machine	lb
N_f	Normal force of both front wheels	lb_f
N_r	Normal force of both rear wheels	lb_f
L	Distance between front wheels and rear wheels	in
L_c	Distance between rear wheels and center of mass	in
h_c	Height of the center of mass	In
F_{Tf}	Tractional force of both front wheels	lb_f
F_{Tr}	Tractional force of both rear wheels	lb_f
F_T	Total tractional force of all wheels	lb_f
F_N	Normal force of payload	lb_f
μ	Coefficient of friction	<i>unitless</i>
θ	Angle of incline plane	°
F_{prop}	Propulsion force	lb_f
F_{rol}	Rolling force	lb_f
$F_{m,v}$	Vertical weight component	lb_f
F_μ	Friction force	lb_f
$F_{inertia}$	Inertia force	lb_f
μ_{rol}	Coefficient of rolling friction (resistance)	<i>unitless</i>
p	Drive screw pitch	rev/in
e	Drive screw efficiency	<i>unitless</i>
T	Torque	$lb\cdot in$
T_{prop}	Propulsion torque	$lb\cdot in$
F_g	Gravitational force	lb_f
q	Load	lb
L	Length	in
r	Radius	in
R	Resultant force	lb_f
I_x	Polar moment of inertia	in^4
σ_{yp}	Yield stress	Ksi
KE	Kinetic energy	$lb_f \cdot in$
G	Modulus of rigidity	Psi
V	Volume	m^3 or in^3
J_L	Load inertia	$lb\cdot in\cdot s^2$
J_{ls}	Lead screw inertia	$lb\cdot in\cdot s^2$
J_m	Motor inertia	$lb\cdot in\cdot s^2$
J_t	Total inertia	$lb\cdot in\cdot s^2$
α	Angular acceleration	rad/s^2
T_f	Friction Torque	$lb\cdot in$
T_{req}	Required Torque	$lb\cdot in$
g	Gravity	m/s^2

1. Introduction

Our goal is to design, fabricate, test, and demonstrate an autonomous bulk material transporter. The transporter needs to follow a predetermined pathway and dump a payload into a collection bin. We need to come up with an effective, creative solution for this task. The transporter will be composed of two main systems: the drive system and the payload delivery system. The drive system needs to autonomously negotiate the several 90° turns and up to 20° inclined ramps of the set path. Once the transporter reaches the end of the pathway, the payload delivery system must offload the lead disks into a container separated by 3" from the ramp structure. The drive system must then return the transporter to the start of the pathway.

Our budget for this project was \$350, which forced us to put a premium on material salvage and ingenuity. Exotic and specialized materials were off the table due to our budget constraints. We also put a premium on simplicity for our system. Dealing with the intricacies and problems of even a simple robot system proved challenging so we attempted to keep our ideas simple, using basic mechanics and gravity's help to achieve our goal.

We have designated our model *Panthra*. Our three original designs were given Thundercat names, from the 1980's children's TV show. Our final model was a combination of *Panthro* and *Tygra*, so we named it *Panthra* as a combination of the two.

A bulk material transporter of this nature is somewhat state of the art. In the application of transporting radioactive lead disks, previous methods have relied on human operators and individually sealing the contaminated disks in bulky packaging. The materials can then be handled and moved with the same methods of standard hazardous cargo. With an autonomous transporter, humans are removed from the equation, saving material used for containment and packaging as well as the reducing contamination risks of handling hazardous materials. Our design was built somewhat on the shoulders of previous year's successes, but this year we saw a much larger load that required new design considerations.

Design Requirements

Objectives & Goals

The objectives and goals of our design are to find a creative solution to the bulk material transporter. We wanted a robot that could achieve all of the high and low level design requirements without looking like the “garbage can on wheels” look of previous years. We also put a priority on speed and power, allowing our machine to make multiple faster runs with a lighter load (3 disks versus 5 or 6).

Ramp Description

Our vehicle needs to transverse through the following ramp and reach the end to offload the lead disks into a collection bin. To do this we need to make sure we can go through the various slopes and turns of the ramp.

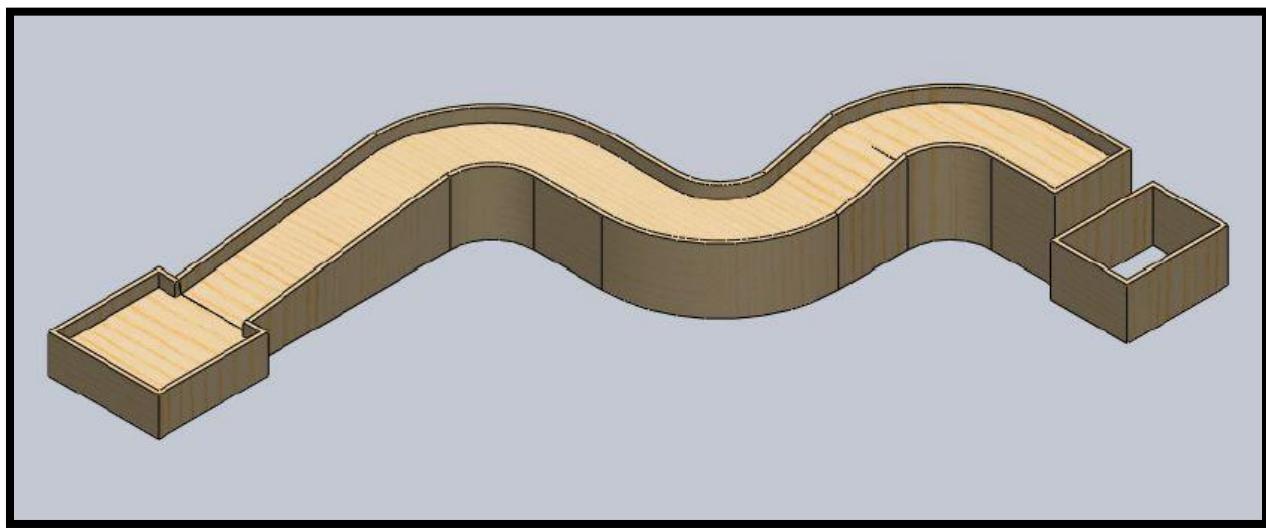


Figure 2: The Spring 2013 Delivery Ramp

Ramp description:

- Made of wooden boards
- Unloading collection bin is 3in. from the ramp.
- Slope of 14° for 16" sloped stretch and 7° for 48" sloped stretch
- Track width is 16" throughout
- Track has a total height change of 10"
- Starting box is 24" by 24"
- Unloading box is 16" by 24"
- Wall height is 3" throughout

Further dimension details are listed in the figure below.

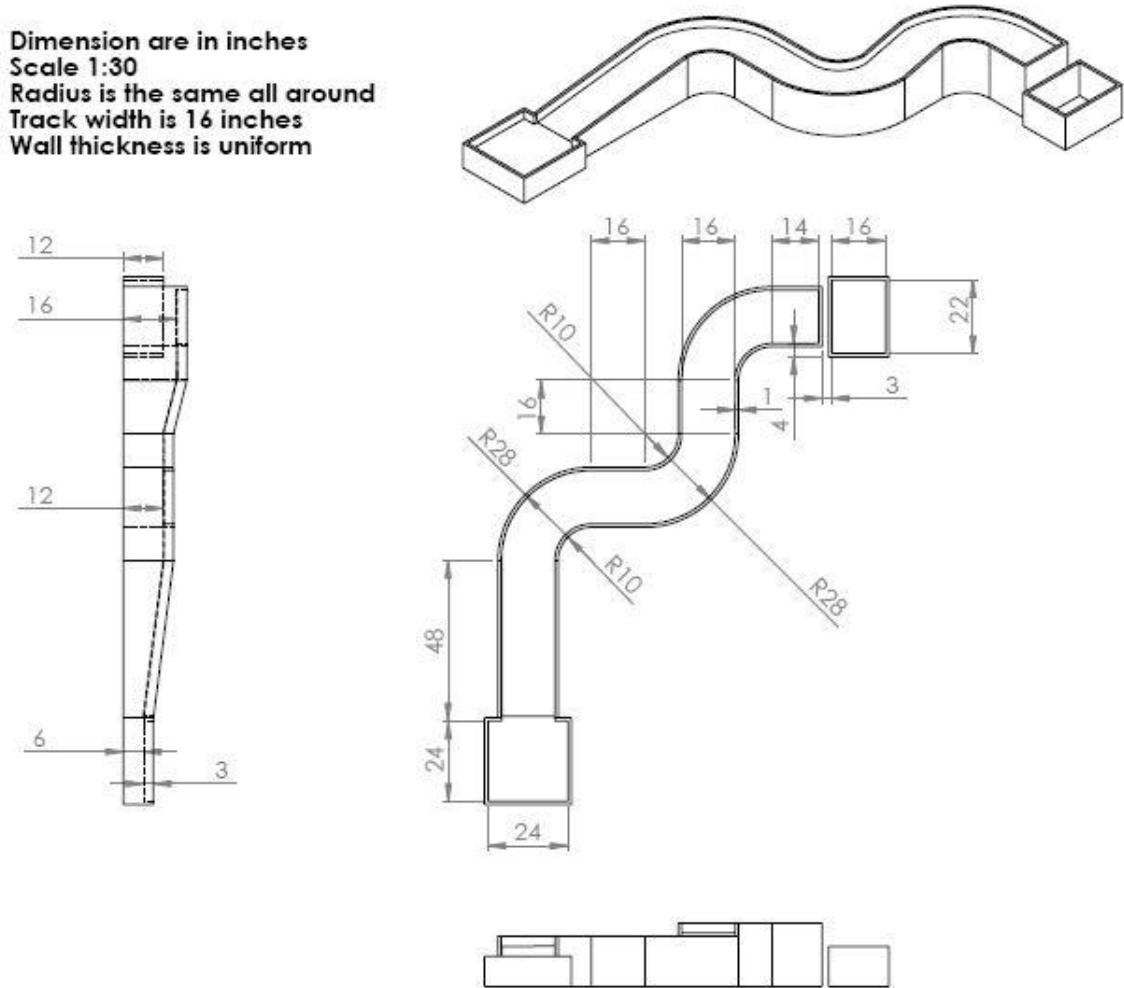


Figure 3: Ramp dimension details [6]

High Level Design Requirements

The High Level Design Requirements are requirements that our team must adhere to. These type of requirements are what base the project, as in our projects low level designs will be based off the needs to satisfy the high level designs. The high level designs are as follows [7]:

1. Device must be no larger than 12x12x16 inches when fully assembled.
2. Vehicle may only be powered using ordinary and readily available rechargeable or disposable batteries. Battery types should not be mixed. Battery packs may be used as long as they comply with the requirements, and can easily be checked by the judges.
3. Disposable battery limit: 10 1.5 V (AAA,AA,C,D) and 3 9 volt batteries, making a total of 13 batteries.
4. Rechargeable battery limit: 12 1.2 V and 3 9 volt, making a total of 15.
5. No other power sources may be used, i.e. hydraulics, combustion and so on.
6. All powered devices must have a clearly labeled and readily accessible emergency kill switch to shut off the device.
7. 1 on/off switch may be installed, and can only be used in the starting/loading area.
8. Budget is 350 \$, no outside money may be used, all receipts must be accounted for.
9. Ramp has set dimensions as stated in “Ramp Description” above, these are predefined and cannot be changed.
10. The payload will consist of a minimum of 3 lead discs, about 6 lb. each. The discs have predefined dimensions, outer diameter of 6”, inner diameter of 25/64” and a thickness of 0.5”.
11. Run time is 10 minutes for the demonstration with a 5 minute set up time. For the competition, run time is 5 minutes and set up time is 2 minutes.

Low Level Design Requirements

The following are our groups design requirements that our team created to guide our decision making during the conceptual phase:

1. Deliver 3 disks per run (approx. 18 lb.)
2. Make a complete run in 45 seconds, requiring a speed of approximately 0.5 – 1 m/s, accounting for maneuvering time.

3. Simplicity in design - using gravity to aid in the delivery systems rather than battery powered sources or complicated sources.
4. Ability to reset payload delivery system for return trip (a bonus if done on the move)
5. Aesthetically appealing (attempt to avoid the trashcan on wheels look of previous years)
6. Use reusable batteries for practice runs to save money, and then use disposable batteries for the final runs and competition.

2. Design Description

Design Concept Development

Through our initial team brainstorming we came up with three individual drive systems and three payload delivery systems. For our drive systems we have the three following:

Three Wheels: For this concept we have two drive wheels in the front with an unpowered omni-directional wheel in the back.

Tracked: This model uses a tank track system. These systems commonly have two motors, but sometimes have four.

Four Wheels: For this concept we have four drive wheels at each corner of the vehicle.

For each of these, no steering system is incorporated. Instead each uses a rotating method to turn where the left and right side drive system (wheels or tracks) spin in opposite directions. This should simplify our drive system and give us the option for 4WD.

The challenge in the payload delivery system is to find a method to transfer the plates across the 4" gap between the ramp and container. For the delivery system we have the following three ideas:

Rear Door/Ramp: In this method, the plates can be stored sideways and the rear door with a bottom hinge descends. Once descended, the door is now a ramp that the plates roll down into the container.

Spindle Arm: In this method, the plates are stored flat, secured with a vertical tube (spindle) running through their middle attached to a horizontal arm. In order to deliver, the arm extends. The spindle then rotates and the plates slide down the spindle into the container.

Retractable/Extending Rear Ramp: In this method, a ramp extends from the vehicle, down which the plates slide or roll. The plates can be secured in place with a rear door with a top hinge.

The initial design concepts are combination of these different drive and payload delivery systems.

Vehicle Type 1: - *Cheetara*- Three Wheeled with Rear Door/Ramp

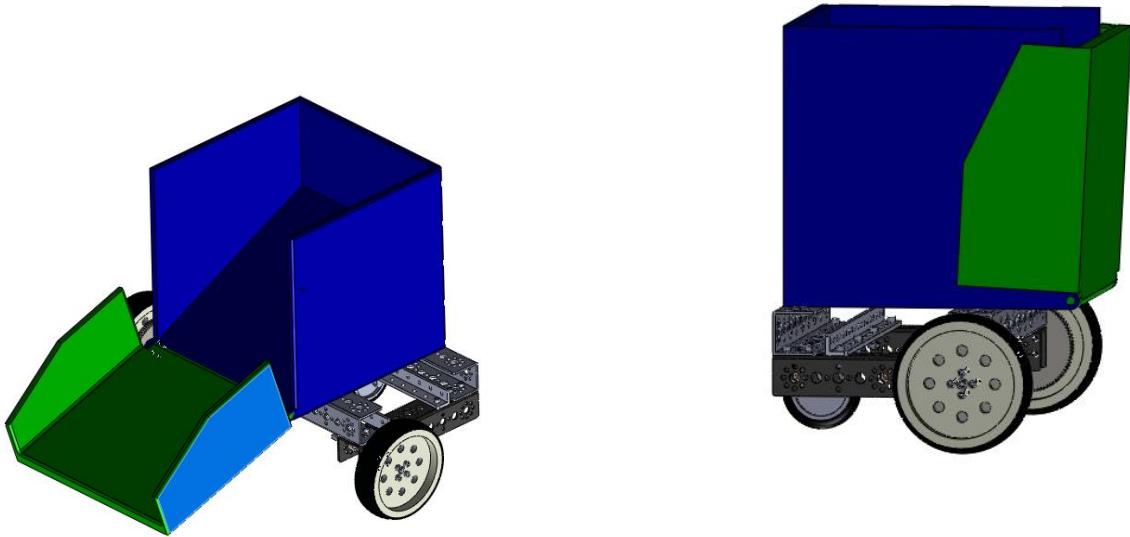
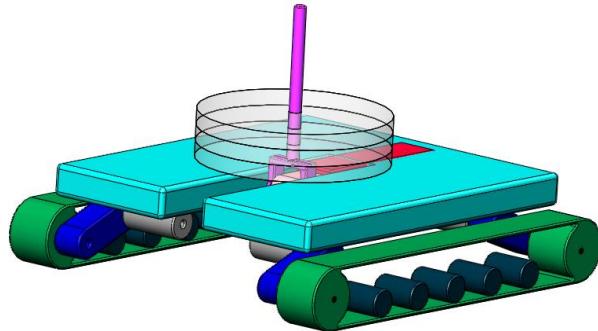


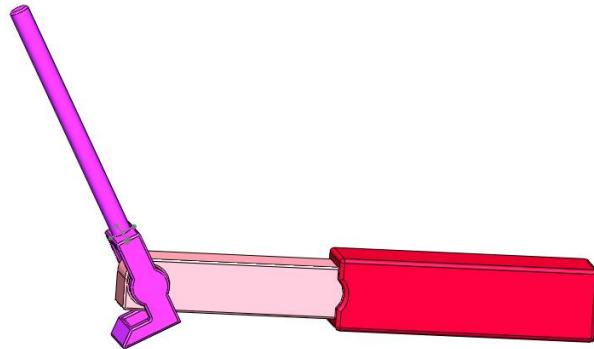
Figure 4: *Cheetara* conceptual design

The advantage of this vehicle lies in its simplicity. Its drive system relies on only two motors with a third non-drive multi-directional wheel. Because of the resulting small vehicle footprint, maneuverability should be improved as well. The payload delivery systems when under load is operated with the help of gravity (door opening downward), using it to help the plates down the ramp.

The disadvantages lie in its payload. With only two motors, the vehicle may not have enough power to deliver plates as quickly as needed. This might be solved with more powerful motors. Another issue with the bottom-hinged door is releasing the locking mechanism while loaded (with plates resting against the door). This may be solved with some creative engineering.

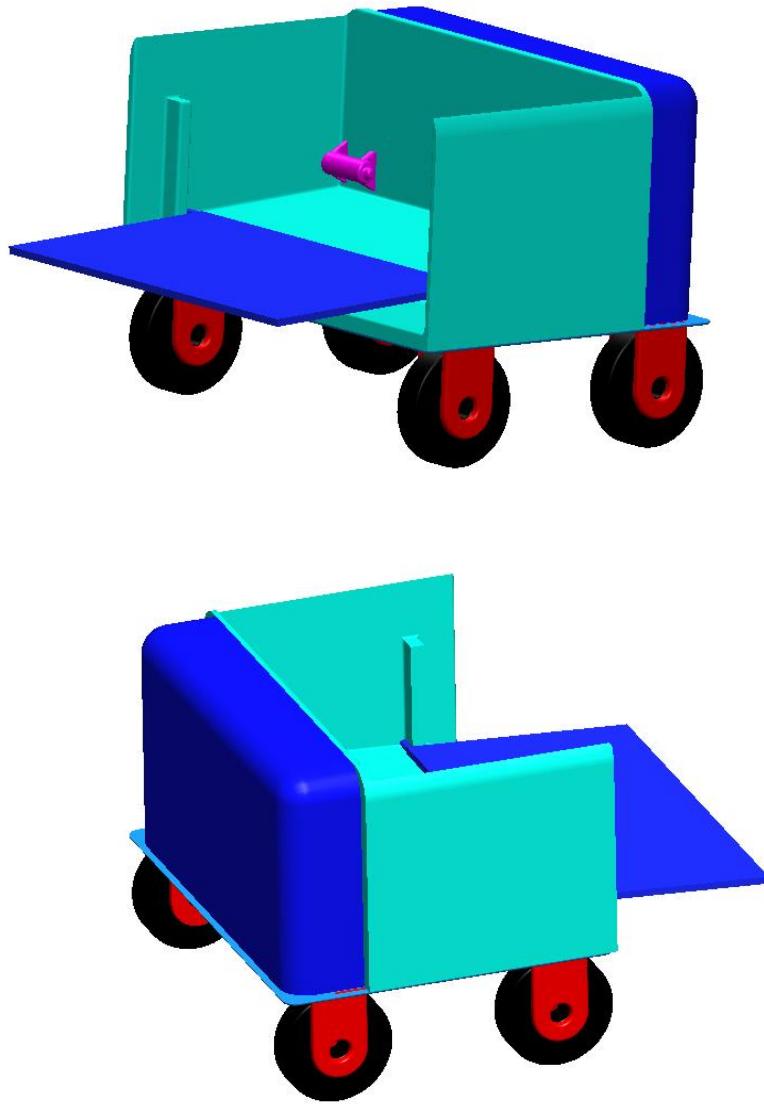
Vehicle Type 2: - *Panthro* - Tracked with Spindle Arm**Figure 5: *Panthro* conceptual design**

The advantage of this vehicle was in its payload and aesthetics. If provided four motors, it should have enough power to deliver a large payload with the high traction rubber tread. Speed may be a decisive factor. The spindle arm greatly improves aesthetics, giving the appearance of a true vehicle transporter and less that of a garbage can with wheels.

**Figure 6: Detail of *Panthro*'s payload delivery system**

The delivery arm is the challenge in this design. The concept seen above has the ability to lock vertically when in the retracted position and then rotate to a downward angle when extended. Appropriate material and shapes that can handle the moment exerted by the three disks will require research and prototyping.

The disadvantage is the complexity. Timing the four motors may add difficulty. Most of the complexity is in the spindle arm along with the obvious counterbalance issues – as the arm extends out the vehicle may tip. This can possibly be solved with an extending pivot from the arm and some engineering ingenuity.

Vehicle Type 3: -*Tygra* - Four Wheeled with Extending Rear Ramp.**Figure 7: *Tygra* conceptual design**

The four-wheeled drive system appeared a safe bet having been the winning choice for previous year's competitions. The payload delivery system only includes the moderately complex retractable ramp. By having a top hinged rear door to secure the plates, the reset of the system is aided by gravity. The design here is shown without the top hinged door, using instead a lock down system to secure the load.

The disadvantage is the lack of aesthetics and some complexity with the retractable ramp. One option is to retract the ramp horizontally into the hull, while another is to rotate back into its initial position (similar to the *Cheetara* design).

Table 1: Concept Design Tree

Sub-function	Carrier	Task	Principle of Evaluation	Weight Factor	Type 1 <i>Cheetara</i>	Type 2 <i>Panthro</i>	Type 3 <i>Tygra</i>
Speed	Drive System	Optimal number of motors	Ability to perform run in <1 minute. Target speed of approx.. 0.5 – 1.0 m/s	0.20	1	3	3
Maneuverability	Frame, Drive System, Steering Method	Optimal form for vehicle.	Ability to maneuver track quickly and efficiently, maintaining sensor tracking as well as turn around.	0.15	3	2	2
Payload	Frame Suspension Motors Battery Pack	Optimal drive system and battery pack and frame.	Number of plates per run, ability to complete multiple runs	0.18	1	3	3
Simplicity	Wheels, Handlebars	Minimal Complexity	Simple methods to move vehicle and deliver payload, ease of troubleshooting	0.11	3	1	2
Creative Evaluation				0.09	1	3	1
Cost Evaluation				0.18	2	2	2
Aesthetic Evaluation				0.09	1	3	2
Weighted Totals				1.0	1.70	2.27	2.29
Order of Merit					3rd	2nd	1st
Notes:							
-Evaluation Marks: 0 - Unacceptable, 1 : Acceptable, 2 : Good, 3 : Optimal							
-See Pair Wise Comparison Chart below for weighted scores.							

Table 2: Pair-Wise Comparison Chart

Subfunction	Speed	Maneuverability	Payload	Creativity	Cost	Aesthetics	Simplicity	Total	Weight	
Speed	5	8	5	8	4	8	6	44	0.20	
Maneuverability	2	5	4	7	4	7	4	33	0.15	
Payload	5	6	5	7	4	7	6	40	0.18	
Creativity	2	3	3	5	2	5	4	20	0.09	
Cost	6	6	6	8	5	8	7	39	0.18	
Aesthetics	2	3	3	5	2	5	5	20	0.09	
Simplicity	4	6	4	3	3	5	5	25	0.11	
[Factor Scale: 10 – each vote counts 2 points]								Totals:	221	1.00

Design Scores: [out of 3]

Weighted Design Scores:

	Type 1	Type 2	Type 3	Type 1	Type 2	Type 3
Speed	1	3	3	0.20	0.60	0.60
Maneuverability	3	2	2	0.45	0.30	0.30
Payload	1	2	3	0.18	0.36	0.54
Creativity	1	3	1	0.09	0.27	0.09
Cost	2	2	2	0.35	0.35	0.35
Aesthetics	1	3	2	0.09	0.27	0.18
Simplicity	3	1	2	0.34	0.11	0.23
				1.70	2.27	2.29

Type 1:	3 wheel (2 powered) vehicle with bottom hinge rear ramp to deliver items.
Type 2:	Tracked vehicle with spindle arm to deliver weights
Type 3:	4 wheel (4 powered) vehicle with retracting rear ramp and top hinge.

Note: Each team member is given two votes to choose their favorite vehicles

Final Design

From our pairwise comparison and analysis, both the *Panthro* and *Tygra* scored very high. We decided on a combination of these two vehicles, designated *Panthra*. It would feature the payload delivery system of the *Panthro*, with the simpler, higher power drive system of the *Tygra*.

Though the spindle arm concept is more complicated than a simple “dump truck” style design, we thought the creativity and aesthetics gave our the design a distinct advantage over the others. The final design of the actual spindle arm has evolved through multiple iterations.

Design Overview

The final design we came up with is called *Panthra* which is the combination of *Panthro* and *Tygra* as described above. The decision was base objectively on design uniqueness and power. The *Panthro*’s spindle arm design was considered unique making it one of our most valued options for the unloading mechanism. The driving system was chosen based on torque power. The four wheel base direct drive provided the most torque without as much friction when compared to the track system. The diagram below shows our final design.

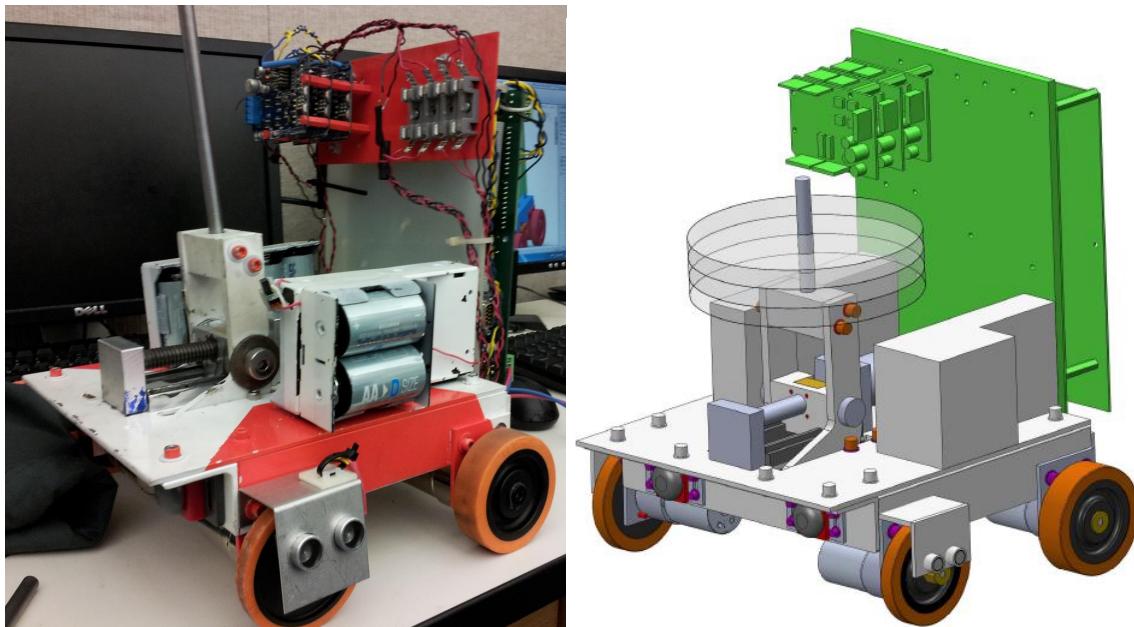


Figure 8: Panhra final design physical design (left) Solidworks model (right).

There are three main systems that help the device operate successfully when tested. There are more subsystems as described later but the main systems are the vehicle drive system, payload delivery system, and electronics and control system. During the motion up the ramp the four motor direct drives provide the necessary torque to drive up the ramp. Once reaching the end the payload delivery mechanism comes into play and unloads the lead disks. This payload delivery mechanism moves down and lets the disk slip past the receiver box edge during unloading. Throughout this process the whole vehicle is under the control of the electronics and control

system. This system collects data from the sensors, process the data using pre-installed labview code, and transmit actions to the corresponding motors. This collection, processing, and transmission of data control the motors speed and direction.

Systems Specifications

The following are our groups actual low level design requirements that are met by our physical transporter:

1. Deliver 3 disks per run. We can do 4 discs per run, but our time and steering is better with 3 discs so we've chosen to stick with that.
2. We make a complete run in about 70 seconds. This includes going up the ramp, delivering the load, turning around, going down the ramp, turning around again, and waiting for the discs to be reloaded. We are hopeful that we can do 4 runs in the 5 minutes we have for the competition (the last run doesn't need to wait for the discs to be reloaded which saves about 10 seconds).
3. Simplicity in design – our delivery arm works perfectly, and we take advantage of gravity (so that our discs slide off the arm without extra force).
4. We raise our delivery arm a little bit before we turn around. The delivery arm rises to its full reset position on the way down the ramp, which saves about 10 seconds.
5. Our robot looks very clean cut, basically a plate on wheels with our spindle arm. Unfortunately our batteries had to be mounted on the top plate which doesn't look as nice as we'd like, but the paint job makes it look more aesthetically appealing.
6. Throughout the quarter we have been using our own personal rechargeable AA batteries so save money. We will be using fresh disposable D batteries for the competition.

Overall System Design

ITEM NO.	PART NUMBER	DESCRIPTION	QTY.
1	1	VEHICLE SUPPORT STRUCTURE	1
2	2	ELECTRONICS AND CONTROLS SYSTEM	1
3	3	VEHICLE DRIVE SYSTEM	1
4	4	PAYOUT DELIVERY SYSTEM	1
5	5	VEHICLE SENSOR ARRAY	1
6	Lead Disk weight 5.75lb	LEAD DISC, 5.75LBS, 6" DIAMETER	3

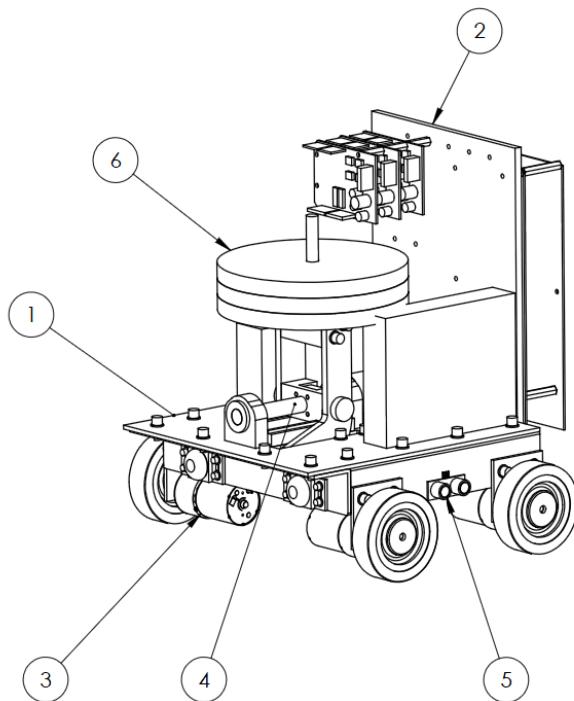


Figure 9: Subassembly overview

Vehicle Drive System (VDS)

The VDS consists of 4 Tetrix Motors and 4 wheels. The two motors on each side will be run in parallel off of one motor driver. One motor driver will then control the left side motors with the other controlling the right side motors. There is no mechanical steering, the vehicle instead utilizes skid steering to turn.

Payload Delivery System (PDS):

The PDS is composed of a lead screw drive system connected to a swinging spindle arm that carries the lead disks horizontally with a vertical shaft through their center holes. To offload, the lead screw moves the device forward, which is lowered steadily through a mechanical cam-roller to a below horizontal position. The retract, the lead screw reverses, the cam-roller action self rights the spindle arm. The curved portion of the spindle arm responsible for the self righting has been designated *The Fang* due to its sharp tooth-like appearance.

Vehicle Support Structure (VSS)

This is the primary load distribution system onto which the other systems are mounted including wheels, motors, sensors, etc.

Electronics and Control System (ECS)

The ECS systems includes the National Instruments SBRio, motor drivers, as well as the battery packs.

Vehicle Sensor Array (VSA)

The VSA includes two front push sensors and two lateral ultrasonic sensors pointed outward from the left and right sides.

Mechanical and Control System Operation

Driving: our drive system results in four powered wheels utilizing skid steering to turn. This requires a small wheel base and front wheels with lower friction. The motors are controlled through motor drivers using digital inputs for direction and pulse width modulation (PWM) for speed.

Lane Control: Our preliminary idea for lane control is utilizing two ultrasonic sensors on the left and right (possibly oriented forward oblique). These sensors have a minimum range of 3cm, but with the current width of our vehicle, we should maintain approximately 6-8 cm between the sides of the vehicle and the lane edge.

Triggering Offload: We plan on using two touch sensors on the front of the vehicle to recognize the end of the track. When the push sensors are compressed the spindle arm will unload the disk. During this process the control systems run the motor until the spindle is position at an angle. The motor will stop its motion when the SB-Rio reads a current from the closed switch the spindle creates at its unloading position. Afterward, we will be turning the vehicle around for the return trip.

Spindle Arm: In this method, the plates are stored flat, secured with a vertical tube (spindle) running through their middle attached to a horizontal arm. The Spindle Arm will extend forward using a power screw and motor. We have worked significantly on an arm design that only requires one linear actuator (all rotation occurs mechanically with gravity). Our current design, utilizing *The Fang*, mechanically eases the spindle into the down position and the uses a cam-roller system to return the spindle into the upright position on retraction. A significant challenge, though, is the difficulty in machining these parts, particularly *The Fang*.

3. Subsystems Design Description

The following is a more detailed description of the Vehicle Subsystems:

1. Vehicle Drive System (VDS)

- a. As mentioned above, vehicle drive system is made up of 4 Tetrix 12V motors with stock

Tetrix gearboxes connected to Bane Bot 2.875" wheel. The motors' output is determined by the Electronics and Control System (ECS).

- b. As shown in the following motor sizing section, the 4 tetrix motors should provide ample power to propel the *Panthra* up the ramp under a 3 disk load in the desired time.

ITEM NO.	PART NUMBER	DESCRIPTION	QTY.
1	3-101	TETRIX DC MOTOR	4
2	3-003	BANEBOOTS 2.875" WHEEL	4
3	3-004	AXLE SET COLLAR	4
4	3-005	MOTOR MOUNT PLATE	4
5	#4-0.75L	#4 SOCKET CAP SCREW	16
6	#4ss-0.25L	#4 SET SCREW	4
7	M3x0.5-0.375L	M3X0.5 SOCKET CAP SCREW	8
8	#4 WASHER	#4 FLAT WASHER	16
9	M3 WASHER	M3 WASHER	8

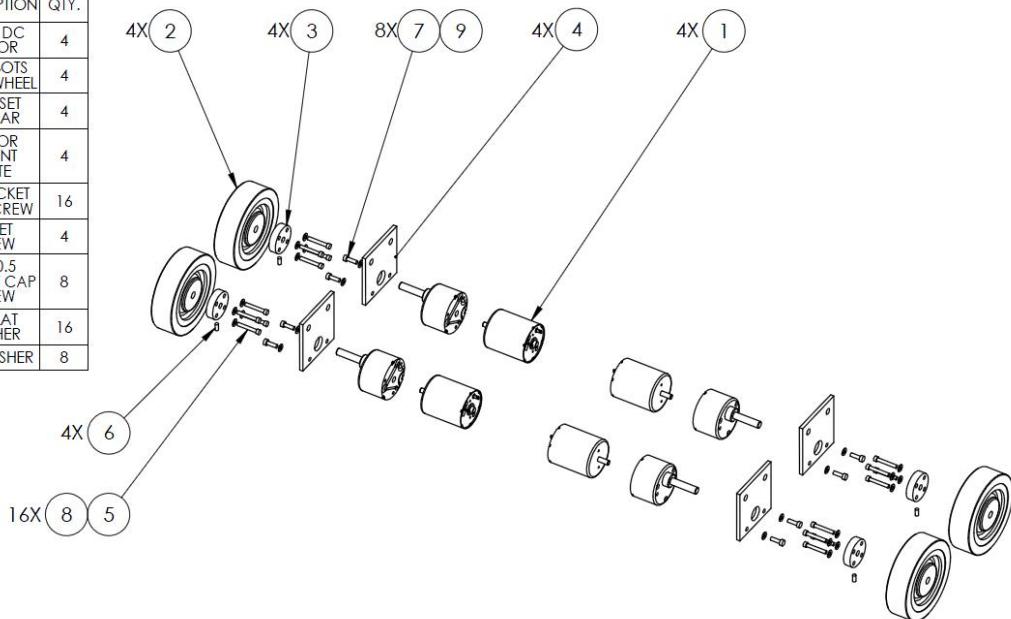


Figure 10: Vehicle Drive System Subassembly Exploded View

2. Payload Delivery System (PDS)

- a. The PDS is made up of a Henkwell 12V DC motor connected to a lead screw. The spindle arm sits on a hinge connected on the lead screw, utilizing *The Fang*, a curved aluminum cam connected to a roller on the body of the vehicle. When the lead screw activates, *the Fang*, lowers the spindle arm in a controlled, static, until the arm is below horizontal. At this point, the lead discs slide off of the spindle arm into the receiving container below.
- b. The spindle arm itself must be a strong enough material to not deform under the load of the lead discs while being narrow enough in diameter to fit into the disc center hole. By lowering the system in a static manner, we prevent kinetic energy from accumulating and having to be absorbed by the spindle arm and *Fang*.

ITEM NO.	PART NUMBER	DESCRIPTION	QTY.
1	4-010	PAYOUT CARRY ARM	2
2	4-011	PAYOUT SPINDLE	1
3	4-012	SPINDLE MOUNT	1
4	#10-0.5L	#10-24 SOCKET CAP SCREW, 0.5" L	4
5	#10 WASHER	#10 FLAT WASHER	4
6	4-102	HENK WELL DC MOTOR ASSEMBLY	1
7	4-005	1/2"-10 LEAD SCREW NUT	1
8	4-009	LEAD SCREW NUT MOUNT	1
9	#4-0.75L	#4 SOCKET CAP SCREW	4
10	4-001	LINEAR ACTUATOR SLIDE	2
11	4-002	DRIVE SCREW BEARING MOUNT	2
12	4-003	DRIVE SCREW BEARING	1
13	4-004	ROLLER MOUNT	2
14	4-006	ROLLER	1
15	4-007	0.5" SHOULDER SCREW	2
16	4-008	0.5" LEAD SCREW, 4" L	1
17	.25-3.0L	0.25" SOCKET CAP SCREW	1
18	.25 WASHER	0.25" WASHER	1
19	0.25-NUT	0.25" HEX NUT	1

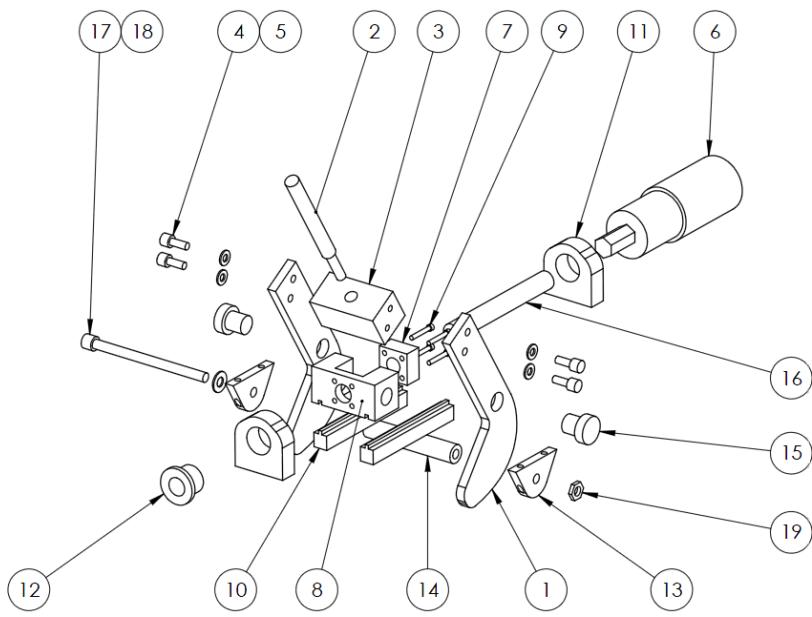


Figure 11: Payload Delivery System Subassembly Exploded View

3. Vehicle Support Structure (VSS)

- The VSS consists of 4 aluminum L beams with a rectangular aluminum plate sitting flat on top.
- The VSS distributes the load of all the mounted systems to maintain a center of gravity within the wheelbase of the vehicle. The VSS must also fit all components within the required dimensions as stated in the higher level requirements.

ITEM NO.	PART NUMBER	DESCRIPTION	QTY.
1	1-001	CHASSIS	1
2	1-002	L-CHANNEL SUPPORT, SIDE	2
3	1-003	L-CHANNEL SUPPORT, FRONT	1
4	1-004	L-CHANNEL SUPPORT, REAR	1
5	#10-0.5L	#10-24 SOCKET CAP SCREW, 0.5" L	12
6	#10-NUT	#10 HEXNUT	12
7	#10 WASHER	#10 FLAT WASHER	12

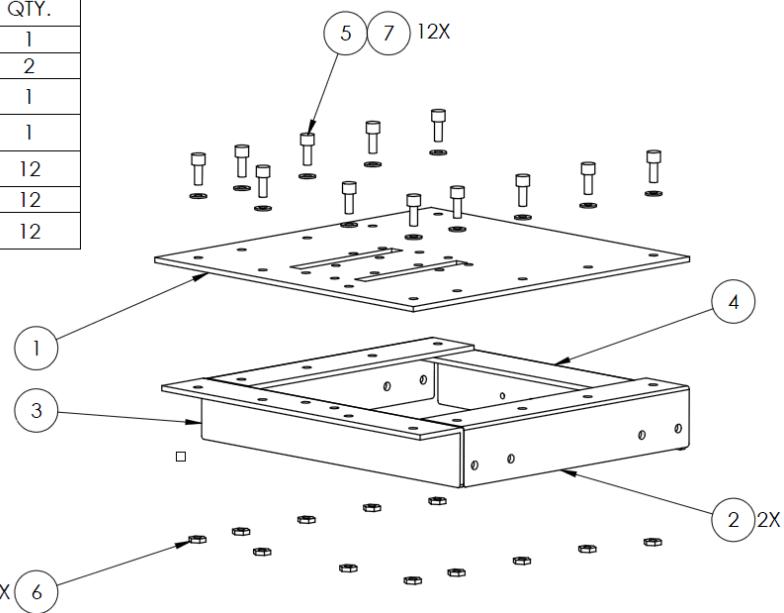


Figure 12: Vehicle Support Structure Subassembly Exploded View

4. Electronics and Control System (ECS)

- The ECS consists of the National Instruments sbRIO, 3 Cytron motor controllers, the sbRIO battery pack, the batteries to power the VDS and PDS, and the voltage regulators to
- The ECS must fit with the dimensional constraints, as well as provide the necessary amperage to drive the VDS.

ITEM NO.	PART NUMBER	DESCRIPTION	QTY.
1	2-101	SB RIO ASSEMBLY	1
2	2-006	SB RIO BATTERY	1
3	2-007	BATTERY PACK, (4) D SIZE	1
4	2-008	BATTERY PACK, (6) D SIZE	1
5	2-009	CYTRON DC MOTOR CONTROLLER	3
6	2-010	BREADBOARD	1
7	#4-STANOFF	HEX STANOFF	6

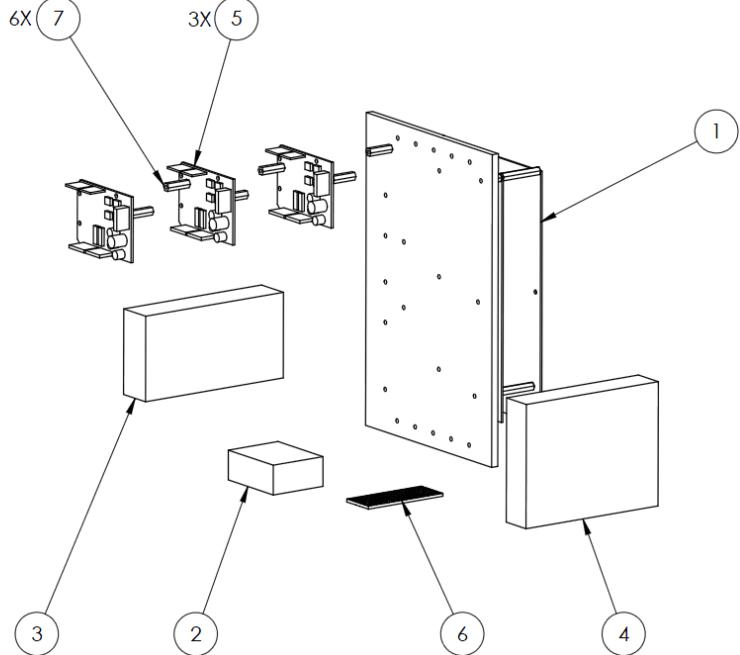


Figure 13: Electronics and Control System Subassembly Exploded View

5. Vehicle Sensor Array (VSA)

- a. The VSA consists of 2 push sensors on the front of the vehicle and 2 ultrasonic sensors – one on each side. When the vehicle reaches the end of the ramp the front push sensors are depressed, triggering offload. To trigger offload, the ultrasonic sensors must also be reading a close distance, indicating the vehicle is in the offload area and the push sensors were not accidentally depressed during a turn.
- b. The ultrasonic sensors must be able to detect a minimum distance of approximately 2 cm, so the lane control system can operate within the 16" wide track. If necessary, the ultrasonic sensors can be mounted closer to the centerline of the vehicle to stay within the minimum sensor range.

ITEM NO.	PART NUMBER	DESCRIPTION	QTY.
1	5-001	VEX BUMPER SENSOR	2
2	5-002	ULTRASONIC SENSOR	2

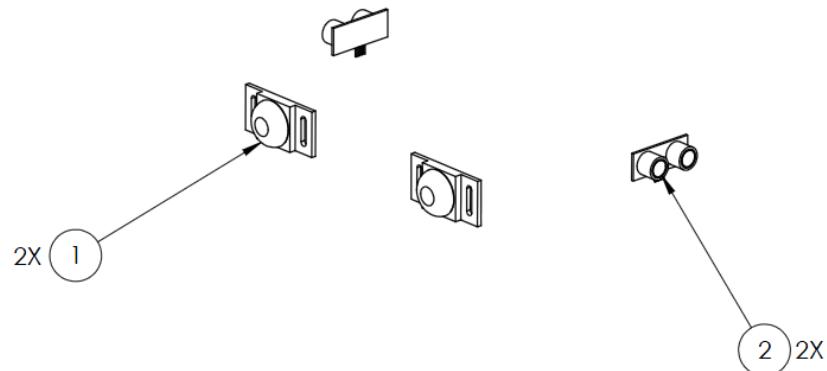


Figure 14: Vehicle Sensor Array Subassembly Exploded View

4. Design Analysis

Power Requirements and Motor Sizing

Part 1: Components Mass

The dimensions of our transporter are 12.0 in long x 12.2 in wide x 11.8 in tall. We are estimating that we will be able to carry 3 disks at a time. Including these in our model, our center of mass is located at (4.29 in, 5.37 in, 6.55 in). These distances are measured from the origin, which is aligned with the bottom of the wheels, the axle of the rear wheels, and the outer edge of the wheel. We have decided to use all wheel drive for our drive system.

Table 3: List of components

Component	Weight (lb)	Multiplier	Total (lb)
Surface Plate	1.57	1	1.57
L-Channel	0.37	2	0.74
Wheel	0.14	4	0.56
Motor	0.38	4	1.52
Electronics	1.88	1	1.88
Batteries	2.94	1	2.94
Electronics Battery	0.63	1	0.63
Back Plate	0.25	1	0.25
Motor Mount	0.05	4	0.2
Bearing	0.08	2	0.16
Drive Nut	0.15	1	0.15
Drive Screw	0.28	1	0.28
Drive Nut Screw	0.02	2	0.04
Drive Bearing Mount	0.08	2	0.16
L-Channel Front	0.07	2	0.14
Payload Spring			
Standoff	0.08	1	0.08
Spindle Arm	0.15	1	0.15
Weight of Transporter		11.45	
Weight of Disks		17.27	
Combined Weight		28.72	

Part 2: Friction Force Calculation

a) Since we do not possess the BaneBot wheels yet, we cannot perform the coefficient of friction test. Therefore we still have to estimate the coefficient of friction. We estimate that μ is around .7, and worst case could be around .5. Therefore we are using $\mu = .6$ for the majority of our calculations.

b to i) Once we know the variable from our CAD model, we can input them into the following excel sheet to solve for normal forces, tractive forces, and the force of gravity along the ramp:

Table 4: Force calculations

Inputs:		Outputs:	(in lb)
$F_m =$	28.72 lb	$N_f =$	12.30
$L =$	5.87 in	$N_r =$	15.51
$L_c =$	4.29 in	$F_{Tf} =$	7.38
$h_c =$	6.55 in	$F_{Tr} =$	9.30
$\theta =$	14.5 deg	$F_T =$	16.68
$\mu =$	0.6	$F_w =$	7.19

Since the total tractive force is greater than the force of gravity along the ramp ($16.16 > 7.19$), our transporter will climb the ramp.

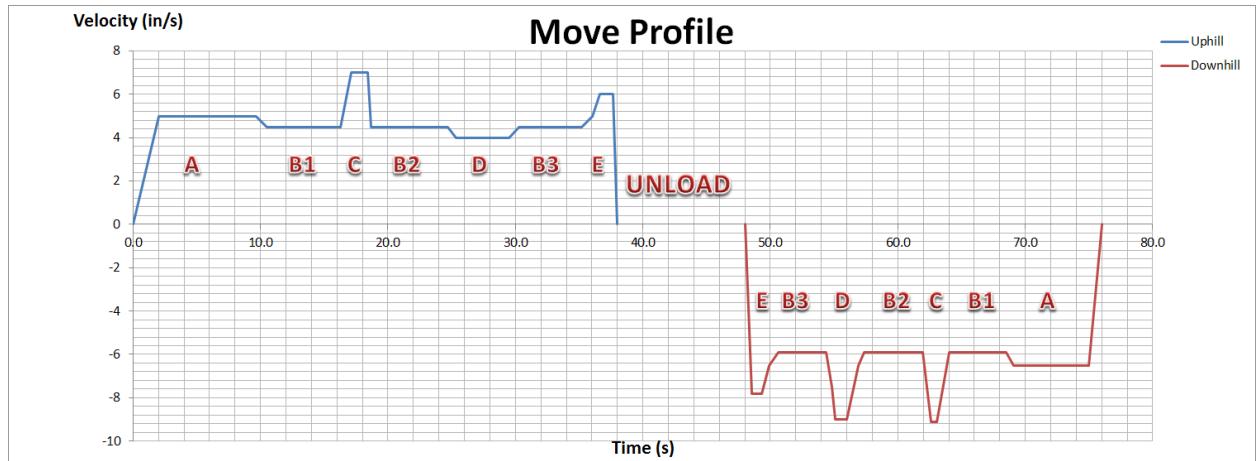
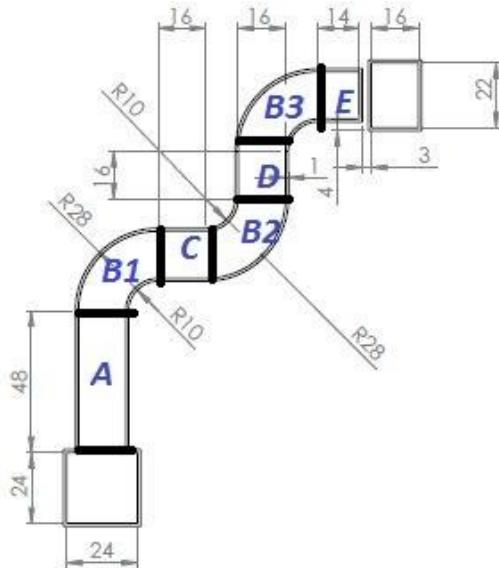
j) Since our drive system is all wheel drive, the minimum coefficient of friction does not depend on transporter design. The minimum coefficient of friction is :

$$\text{(eq. 4-1)} \mu = \tan(\theta)$$

The largest ramp angle is 14.5 degrees, so the minimum coefficient of friction is .26.

Part 3: Move/Velocity Profile

a)

**Figure 15: Round-trip move profile****Figure 16: Labeling of track used in move profile**

b)

Table 5: Maximum velocities of move profile

Section	Uphill (in/s)	Downhill (in/s)
A	5	-6.5
B	4.5	-5.9
C	7	-9.1
D	4	-9
E	6	-7.8

c)

$$(eq. 4-2) \beta_f = \frac{L_c * \cos(\theta) - h_c * \sin(\theta)}{L}$$

$$(eq. 4-3) \beta_r = \frac{(L - L_c) * \cos(\theta) + h_c * \sin(\theta)}{L}$$

Table 6: Beta for uphill

Section	β_f	β_r	θ
A	0.587	0.413	7.1°
B,C,E	0.731	0.269	0°
D	0.439	0.561	14°

Table 7: Beta for downhill

Section	β_f	β_r	θ
A	0.159	0.841	7.1°
B,C,E	0.274	0.726	0°
D	0.044	0.956	14°

Note: When going downhill, the rear wheels are the lead wheels, as it is moving backwards.

d) The total propulsion force is:

$$(eq. 4-4) F_{prop} = F_m + F_T + F_{rol} + F_{Inertia}$$

$$(eq. 4-5) F_{Inertia} = M * a$$

$$(eq. 4-6) F_{rol} = \mu_{rol} * mg \cos(\theta)$$

$$(eq. 4-7) F_f = \mu * mg \cos(\theta)$$

$$(eq. 4-8) F_{m,v} = mg \sin(\theta)$$

The units are in lbf.

Table 8: Required propulsion force for uphill

Section	$F_{Inertia}$	F_{rol}	F_f	$F_{m,v}$	F_{prop}
A	0.186	0.570	17.100	3.550	21.406
B,C,E	0.232	0.574	17.232	0.000	18.038
D	0	0.557	16.72	6.948	24.225

Inertia acceleration was estimated using the assumption of constant acceleration for changes in velocity. To estimate, acceleration is taken as the velocity change over a section divided by the time it takes to cross the section. Rolling friction was taken as a car tire on asphalt, which is a decent estimate.

Part 4: Power Requirement Calculations

a) The following is the vehicle data used in the torque calculations. Propulsion Force was calculated in our previous report:

Table 9: Torque calculations data

Wheel Make	BaneBots 20A Tread
Diameter:	$2\frac{7}{8}''$
Width:	0.8"
Estimated μ_s :	0.7
WheelBase	5.87in (L), 9.16 in (W)
Rear Wheel dist. To CG	
horizontal:	4.29 in
vertical:	6.55 in
Estimated μ_s	0.7
Ramp Slopes	
A: 7.125°	21.406 lb
B, C, E: 0°	18.038 lb
D: 14.5°	24.225 lb

b) To calculate the required propulsion torque for each section we use the following equation:

$$\text{(eq. 4-9)} \quad T_{\text{Propulsion}} = F_{\text{Propulsion}} \times r_{\text{Drivewheels}}$$

$$T_{\text{Propulsion}} = 24.225 \text{ lb} \times 1.438 \text{ in}$$

$$T_{\text{Propulsion}} = 34.84 \text{ lb} \cdot \text{in} = 40.18 \text{ kgf} \cdot \text{cm}$$

(Note that $1.0 \text{ kgf} \cdot \text{cm} = 0.867 \text{ lb} \cdot \text{in}$.)

For each ramp section we get the following, with the minimum required highlighted:

Table 10: Required propulsion torque

Ramp Section	$T_{\text{Propulsion}}$	$T_{\text{Propulsion}}$
A	30.78 lb · in	35.50 kgf · cm
B, C, E	24.94 lb · in	29.92 kgf · cm
D	34.84 lb · in	40.18 kgf · cm

Part 5: Torque Calculation

We decided to build our robot with the direct drive system (note: Tetrix motor has an internal gearbox, but information is not given so we assume that the output torque provided by the company have already taken the gearbox into account). Below is the collection of data for calculating the required torque for our robot. To find the required torque we used the following

torque equation:

$$(eq. 4-10) \quad T_{req} = J_t \alpha_{acc} + T_f = (J_l + J_m) \alpha_{acc} + T_f$$

We first have to find the individual moment of inertia before we can find the T_{req} . The data of the inertia and T_f calculation are listed in color code

Table 11: Required torque for direct drive

J _L	J _m	α	T _f	T _{req}
3.73E-04	1.33E-03	2.10	23.94	23.94
lb-in-s ²	lb-in-s ²	rad/s ²	lb-in	lb-in

Table 12: Load inertia

J _L	3.73E-04	lb-in-s ²
Weight_wheel	0.14	lb
R_wheel	1.435	in
G	386	in/s ²

Table 13: Motor inertia

J _m	1.33E-03	lb-in-s ²
m_shaft	2.22E-02	lb
R_shaft	0.12	in
p_density	0.28	lb/in ³
V_volume	7.92E-02	in ³
Shaft_length	1.75	in

Table 14: Friction Torque

T _f	0.22	lb-in
T _{f(max)}	23.94	lb-in
F _s	16.68	lb-in-s ²
Robot Weight	28.72	lb
μ (Friction)	0.6	unitless
R_wheel	1.435	in

From our previous calculation with 4 Tetrix Motor we achieve a 40.65 lb-in of torque. This means that the require torque 23.94 lb-in is less than the motor drive torque. In conclusion this means that the vehicle can move up the ramp without significant problem since the require torque is about 1/2 the drive torque.

$$(eq. 4-11) \quad T_{req} = 23.94 \text{ lb-in} < T_{drive_Torque} = 40.65 \text{ lb-in}$$

Part 6: Unloading Mechanisms

The payload, as previously described, uses a linear actuator to move the payload forward to a point at which a mechanism touches the sidewall of the ramp. At this point, the mechanism which

holds the payload tips forward, and allows the payload disks to slide off into the collection box. The linear tipping mechanism uses gravity and some form of angular damper, but the linear actuator requires power. The estimated power is derived below.

The *Panhra* vehicle must transport a minimum of 3 disks (18lb) up the ramp. As shown in the diagram below, the drive screw (linear actuator) will only need to move the payload horizontally, so the only load on the actuator will be due to friction in the drive screw.

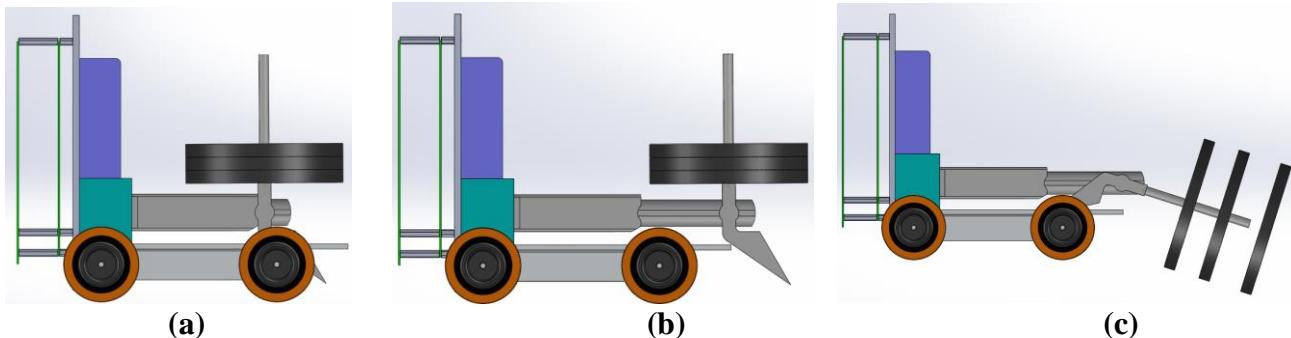


Figure 17: a,b,c: (a) transport configuration, (b) delivery configuration with extended actuator, (c) delivery configuration with extended actuator and tipped payload

In the above figures, the vehicle is shown in (a) the transport configuration, (b) the delivery configuration with extended actuator, and (c) the delivery configuration with extended actuator and tipped payload. The drive screw must provide the force to extend the payload horizontally for approximately 3". For a $\frac{1}{2}$ " – 10 drive screw, the lead angle is given by

$$(eq. 4-12) \lambda = \tan^{-1}(\frac{1}{10}/0.5\pi) = \tan^{-1}(2/10\pi) = 3.64^\circ$$

For a drive screw in various nut materials, kinetic friction coefficients are in the range of 0.11-0.17 if oiled. A friction coefficient of 0.15 will be assumed. For a stainless steel ACME drive screw with a metallic nut, common efficiencies are in the range of 35-55%. The torque required to overcome friction in moving a load by a drive screw is given by:

$$(eq. 4-13) T = \frac{F_\mu}{2\pi pe} = \frac{\mu F_N}{2\pi pe}$$

For a normal force of 18lbf, a pitch of 10 (assuming a $\frac{1}{2}$ "-10 drive screw), and an efficiency of 45%, the required torque is 0.0955 in-lb.

Additionally, some torque is required to overcome the rotational inertia of the lead screw. For a 5 inch steel lead screw with $\frac{1}{2}$ " diameter

$$(eq. 4-14) T = J = \frac{\pi L \rho R^4}{2} = 0.1424 \text{ in-lb}$$

Thus, the final torque required is

$$T = 0.0955 \text{ in-lb} + 0.1424 \text{ in-lb} = 0.2378 \text{ in-lb}$$

If the same motor is used as for the wheels (TETRIX DC Gear Motor, PN 739023), a stall torque of 20kg-cm = 17.36in-lb can be provided, which is far more than enough. At peak efficiency, the

motor supplies 4.85 kg-cm of torque, which translates to 4.21 in-lb, at 128 RPM. If the motor directly drives the lead screw, this would translate to a linear speed of 0.213 in/sec. If the torque curve was provided in an equation form or as an array of values, the differential equation of motion for this problem could be solved. Based on this information alone, however, we can reason that the motor will be able to move the payload in near 15 seconds.

Spindle Arm Stress Analysis:

The following is a preliminary analysis for the viability of the spindle arm delivery system. As the spindle rotates into the downward delivery position large impact forces can be generated. Also, as the weights slide outward on the arm towards the receptacle, large forces and moments are generated on the rod. For these calculations I have made the spindle 1/16th inch smaller than the hole in the lead disk.

The first analysis is finding the stresses generated on the rod angled θ degrees below horizontal with the weights at the end. To simplify the model, we will maintain standard horizontal and vertical axis, while taking the gravitational force as $F_G \cos \theta$.

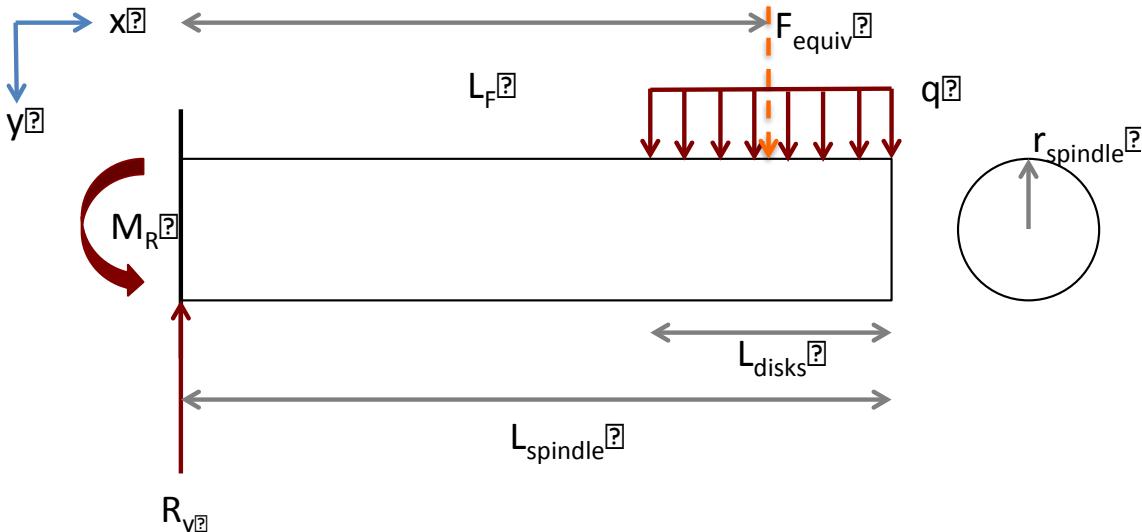


Figure 18: Free body diagram of spindle arm

$$(eq. 4-15) \text{ Load: } q = \frac{F_G \cos \theta}{L_{\text{disks}}}$$

Table 15: Properties of spindle arm

F_G	18 lb	Steel, High Strength ASTM-A242	
F_{equiv}	$18 \cos \theta$ lb	Yield Strength:	
Q	$10.67 \frac{\text{lbs}}{\text{in}}$	Tension	50ksi
L_{disks}	1.5 in	Shear	30 ksi

L_{spindle}	6.0 in	Modulus of Rigidity (G):	11.5E6 psi
L_F	5.25 in		
r_{spindle}	21/128 in		
I_x	5.690E-4 in ⁴		
V	0.1268 in ³		

By balancing moments we get that:

$$(\text{eq. 4-16}) M_R = F_{\text{equiv}} \cdot (L_{\text{spindle}} - L_{\text{disks}}/2)$$

$$M_R = 84.02 \cos \theta \text{ lb}\cdot\text{in}$$

Solving for max stress:

$$(\text{eq. 4-17}) \sigma_{\max} = \frac{M_R r_{\text{spindle}}}{I_x}$$

$$\sigma_{\max} = 24.2 \cos \theta \text{ ksi}$$

We then find our safety factor, if using High Strength Steel ASTM-A242:

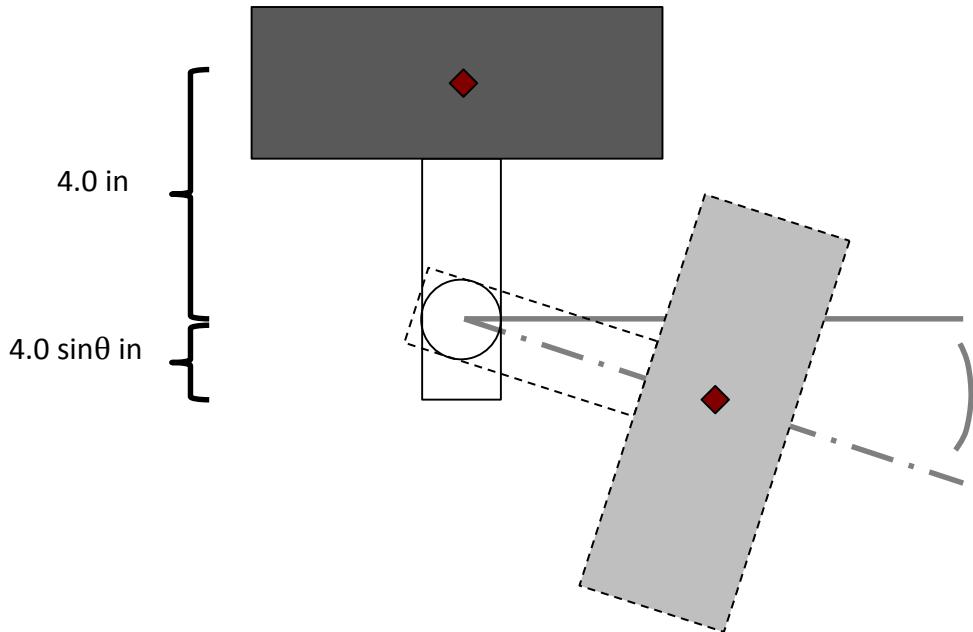
$$(\text{eq. 4-18}) \frac{\sigma_{\text{yp}}}{n} = \sqrt{\sigma_{\max}^2 + 4\tau_{xy}^2}$$

$$n = \frac{1.24}{\cos \theta}$$

This analysis puts us under the yield strength of High Strength Steel with a factor of safety of 1.24. This would be with the spindle horizontal. By increasing the angle below horizontal we would expect less stress and a higher factor of safety.

To analyze the initial impact of the rotating spindle we use shear impact:

$$(\text{eq. 4-19}) \tau_{\text{impact}} = 2 \sqrt{\frac{KE \cdot G}{V}}$$

**Figure 19: Rotation of Spindle Arm**

We computed the Kinetic Energy as the energy transformed from Potential Energy from the lead disks' initial height, to its final height.

$$\text{(eq. 4-20) } KE = 18 \text{ lbs} \cdot 4 \text{ in} (1 + \sin \theta)$$

To simplify this analysis, we'll use $\theta = 30^\circ$

$$KE = 108 \text{ lb} \cdot \text{in}$$

This results in a shear impact of:

$$\text{(eq. 4-21) } \tau_{impact} = 198 \text{ ksi}$$

This value is much too high for the steel spindle to absorb, but also represents a worst case scenario of the lead disks freely falling into the down position. The process could still be viable by instituting some method of dampening, or a mechanical method to ease the spindle into the down position.

Lead Screw Calculations:

For our unloading mechanism we decided to use the lead screw mechanism to push our load to the edge of the robot and let gravity unload the lead disk. To make sure it actually work we performed some basic calculation. The governing equation we use for this calculation is:

$$\text{(eq. 4-22) } T_{req} = J_t \alpha_{acc} + T_f = (J_l + J_{LS} + J_m) \alpha_{acc} + T_f$$

We first have to find the individual moment of inertia before we can find the T_{req} . The data of the inertia and T_f calculation are listed in color code.

Table 16: Required torque for lead screw, $\mu = 0.15$

J _L	J _{LS}	J _M	alpha	T _f	T _{req}
4.6382E-05	0.00875	0.00133	40	0.21092	0.615976
lb-in-s ²	lb-in-s ²	lb-in-s ²	rad/sec	lb-in-s ²	lb-in-s ²

Assumed μ (Friction) to be 0.15 (steel on steel lubricated)

Table 17: Required torque for lead screw, $\mu = 0.58$

J _L	J _{LS}	J _M	alpha	T _f	T _{req}
4.6382E-05	0.00875	0.00133	40	0.815558	1.220614
lb-in-s ²	lb-in-s ²	lb-in-s ²	rad/sec	lb-in-s ²	lb-in-s ²

Assumed μ (Friction) to be 0.58 (steel on steel lubricated)

Table 18: Load inertia and lead screw inertia

J _L	4.64E-05	lb-in-s ²	J _{LS}	8.75E-03	lb-in-s ²
Weight_load	17.67	lb	mass	0.28	lb
P (pitch)	5	rev/in	radius	0.25	in

Table 19: Friction torque for two different μ' 's

T _f	2.11E-01	lb-in-s ²	T _f	8.16E-01	lb-in-s ²
μ (Friction)	<u>0.15</u>	Unitless	μ (Friction)	<u>0.58</u>	unitless
Weight_load	17.67	lb	Weight_load	17.67	lb
e (efficiency)	0.4	Unitless	e (efficiency)	0.4	Unitless

It turns out that the T_{req} is quite small. Again if we choose to use the Tetrix Motor for this screw as well it will perform fine in general since the drive torque is significantly higher than the required torque for the unloading mechanism. Note: We choose the 0.816 lb-in since this will be the max torque the motor will have to go against in the worst case scenario.

$$(eq. 4-23) T_{req} = 0.816 \text{ lb-in} < T_{Drive_Torque} = 10.16 \text{ lb-in}$$

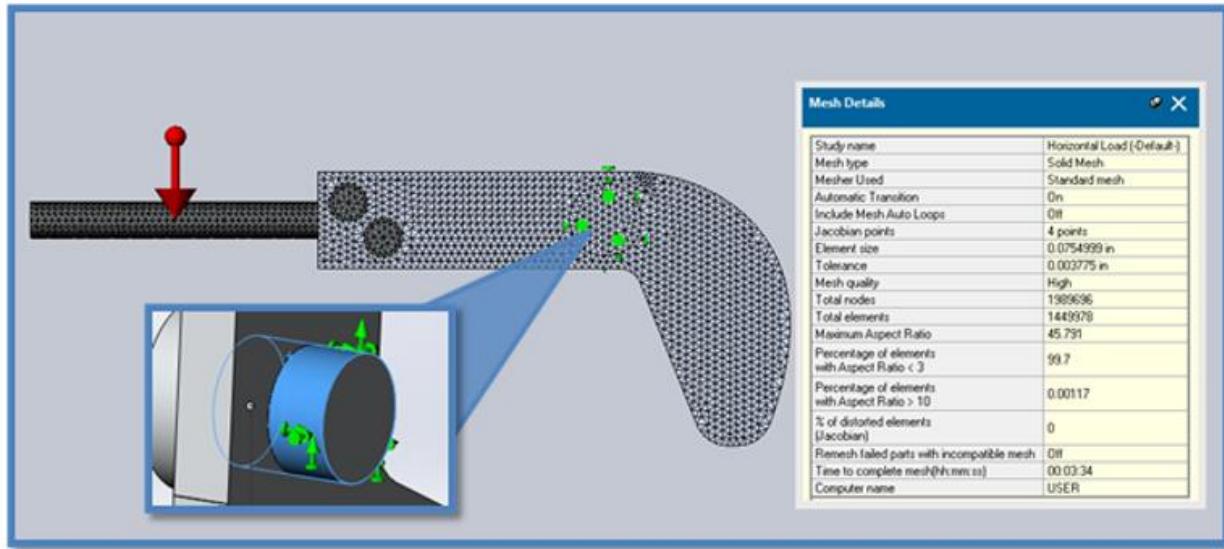
FEM-Analysis of PDS

To verify that the delivery arm can withstand the load we performed an FEA. In our analysis we chose to do an analysis when the PDS is horizontal. During its horizontal position the PDS experienced the most stress and torque. Due to this condition, we decided to perform a test during this position. Any other positions we believe should experience less stress or torque because the weight of the disk is not perpendicular to the loading arm; therefore, resulting in a lower torque.

Table 20: PDS assumptions for FEA

Analysis type	Static Analysis
6 Disks	Place until it reach the end of the rod (Not shown in diagram)
Mesh type	Solid Mesh
Solver type	FFEPlus
Incompatible bonding options	Automatic
Fixtures	Screw mount to drive screw (Green)
Gravity	Y-direction both direction has same result (direction in red)

In this analysis we chose the horizontal position. Our PDS is symmetric making it have similar stress on both sides so applying gravity in either y-direction makes no difference in magnitude of stress, displacement, or FOS. As shown below the fixtures are chosen there because it's the only possible fixtures and point of contact connected to the moving block driven by the lead screw.

**Figure 20: Mesh and fixtures detail. (Fixtures are green and gravity direction in red)**

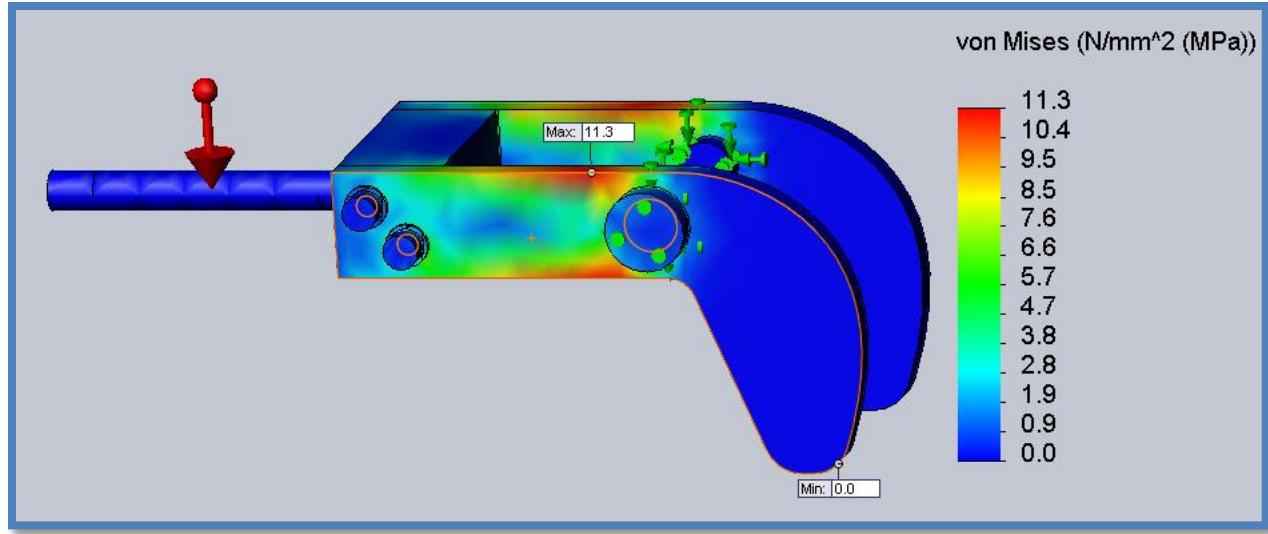


Figure 21: PDS stress analysis during horizontal position.

From the stress analysis there are a few materials that are present and playing a major role in the FEM analysis. The material are Carbon Steel and Al 6061-T6 having a yield stress of 220 MPa and 275 MPa respectively. As we expect both yield stress are greater than the stress supplied by the load. This means that the PDS is safe from failure.

When observing the FEA for displacement the largest displacement is about 0.1mm. In cases like this the results does not amount to much displacement as we would expect since the stress has not approach anywhere near the yield strength.

The Factor of Safety (FOS) test still has a FOS over 19 with 6 disks on. Even during this case the PDS provide prove that we could support up to six disks. In our current situation there is a chance only 3 or 4 disks will be driven up during each run due to battery power.

The only results that these FEA fail to provide detail information on are the joints that are threaded. Due to insufficient information at the joints the FEA cannot provide us with that information. So we would have to perform an actual physical test. During the actual physical test the PDS was able to hold the disk perfectly fine satisfying the requirements to hold the disks without failures.

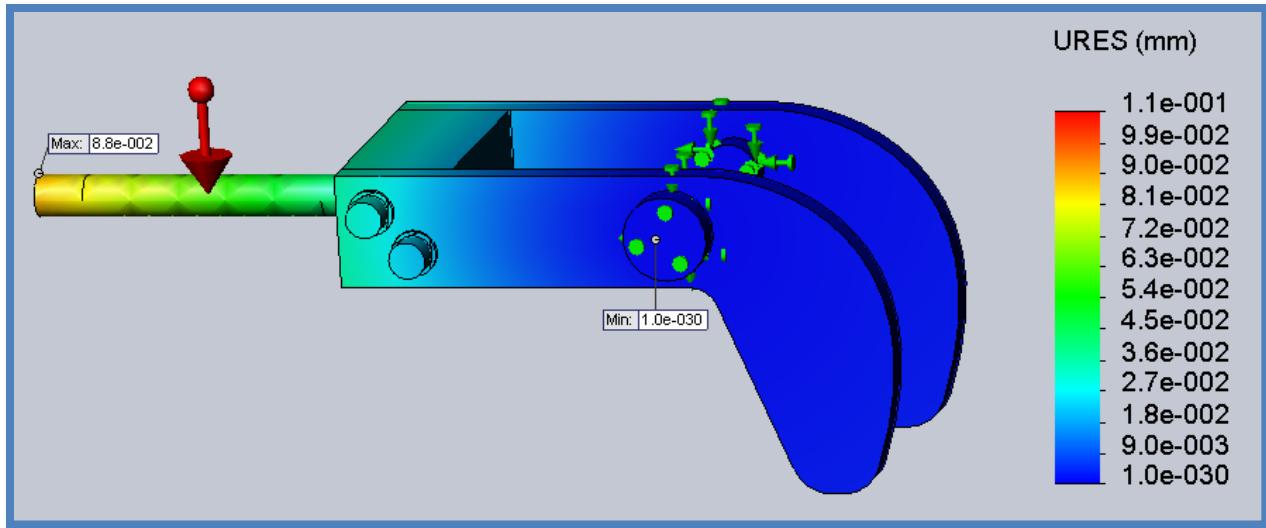


Figure 22: PDS displacement analysis during horizontal position.

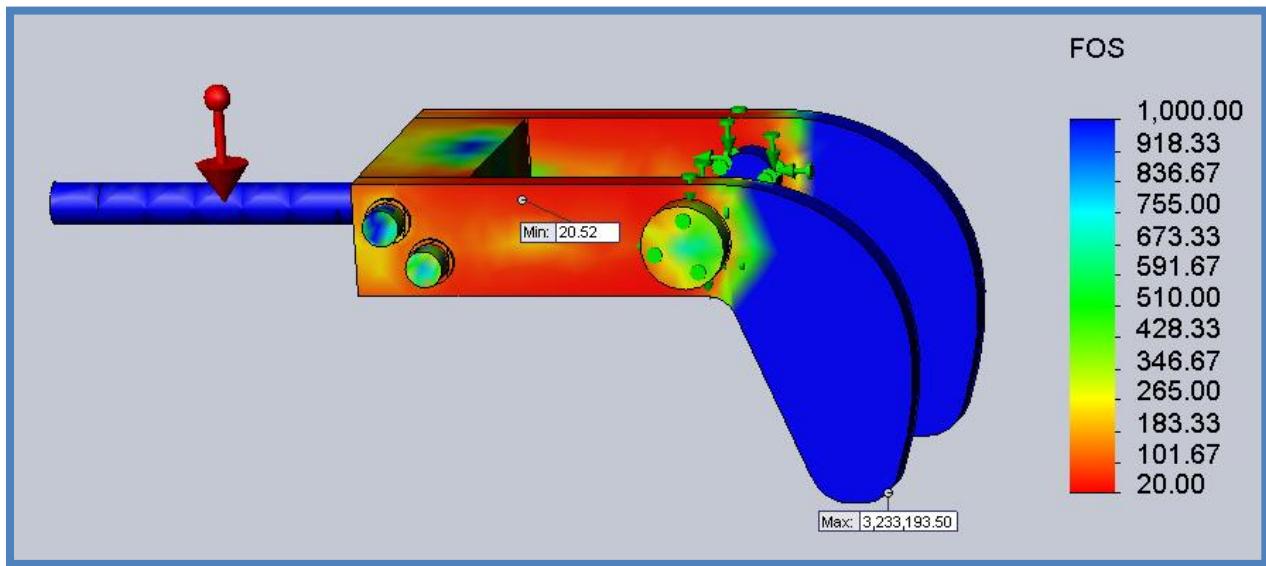


Figure 23: PDS FOS during horizontal position.

5. Control System Design

The following is a more detailed description of the Electronics and Control System (ECS) Design:

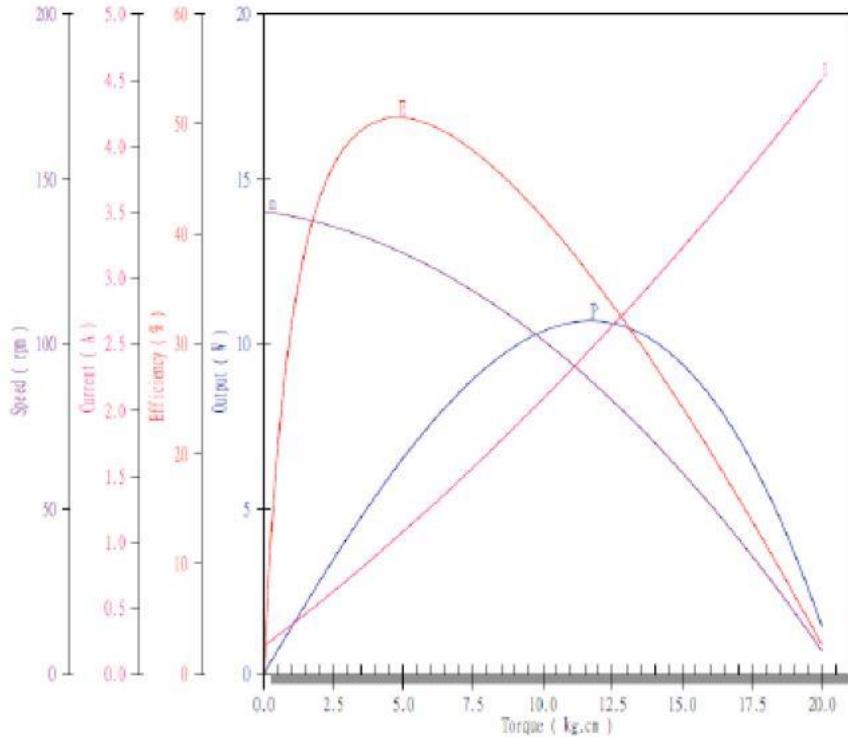
Motor Selection

Motor Details

We have selected the following motor as a candidate: Tetrix DC Drive Motor

Table 21: Tetrix DC Drive Motor Specs

Test Voltage:	12 V DC
Gear Ratio:	1:52
At No Load	
Speed:	146 RPM
Current:	0.17 Amps
At Stall Extrapolation	
Torque	20 kgf · cm
Current	4.55 Amps
At Max Efficiency	
Efficiency	50%
Speed	128 RPM
Torque	4.85 kgf · cm
Current	1.05 Amps
Output	6.38 Watts
At Max Output	
Efficiency	36.22 %
Speed	89 RPM
Torque	11.72 kgf · cm
Current	2.46 Amps
Output	10.7 Watts

**Figure 24: Tetrix DC Motor Performance Curve**

a) Relying on 4 Tetrix Motors (1 powering each wheel), we could theoretically achieve a maximum output torque of:

$$(eq. 5-1) T_{Propulsion,Max} = 4 \times 11.72 \text{ kgf} \cdot \text{cm} = 46.88 \text{ kgf} \cdot \text{cm} \text{ or } 40.64 \text{ lb} \cdot \text{in}$$

Drive Torque: Our current design does not utilize any gearing between the motor and wheels.

b) Running the motors at max efficiency will not result in the necessary output torque for the steepest ramp section. From the plot, an output torque of 10.05 kgf · cm (our minimum required Torque, divided by 4) will result in approximately 40% efficiency. With 50% being the motor's max attainable efficiency, this is reasonable. Running the 4 motors at a 10 kgf · cm at 80 RPM will result in an output of 10 Watts, with a motor efficiency of 40%.

c) Motor circuit diagram are provide in later section.

Motor Driver Selection

The motor drive we chose is: 10A DC Motor Driver Arduino Shield 7-15V

From Plot for the Tetrix Motor:

$$(eq. 5-2) K_T = \frac{T_{MAX}}{I_{MAX}} = \frac{20 \text{ kg} \cdot \text{cm}}{4.55 \text{ A}} = 4.396 \frac{\text{kg} \cdot \text{cm}}{\text{A}}$$

Using Propulsion Torque per Motor = $46.88 \text{ kg} \cdot \text{cm} / 4 = 11.72 \text{ kg} \cdot \text{cm}$

$$(eq. 5-3) I_{PROP} = \frac{T_{PROP}}{K_T} = \frac{11.72 \text{ kg} \cdot \text{cm}}{4.396 \frac{\text{kg} \cdot \text{cm}}{\text{A}}} = 2.67 \text{ A}$$

Using Acceleration Torque per Motor = $5.985 \text{ kg} \cdot \text{cm}$

$$(eq. 5-4) I_{ACCEL} = \frac{T_{ACCEL}}{K_T} = \frac{5.985 \text{ kg} \cdot \text{cm}}{4.396 \frac{\text{kg} \cdot \text{cm}}{\text{A}}} = 1.361 \text{ A}$$

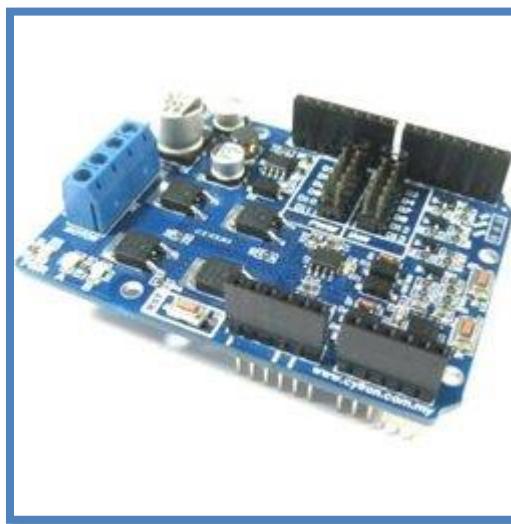


Figure 25: Motor Driver

Price: \$19.06
 Peak Load: 15A
 Continuous Load: 10A
 Operation: PWM input for speed and DI for direction

This motor driver can operate at 10A continuous load; therefore, making it a safe candidate for two motors driven in parallel.

Unloading Mechanism

Motor Details

a)

We have selected the following motor as a candidate for the PDS:
 Hennkwell 12V DC Motor

Table 22: Hennkwell 12V DC Motor Specs

Model	Rated Voltage	Reduction Ratio	1/264
2100	12V	No Load Speed(rpm)	38
		No Load Current (mA)	1000
		Rated Torque (kg-cm)	12.00
		Rated Speed (rpm)	36.5
		Rated Current (mA)	5000
9100	24V	No Load Speed(rpm)	36
		No Load Current (mA)	440
		Rated Torque (kg-cm)	12.00
		Rated Speed (rpm)	34.5
		Rated Current (mA)	770

PK32K3SP2100-264 (12V)

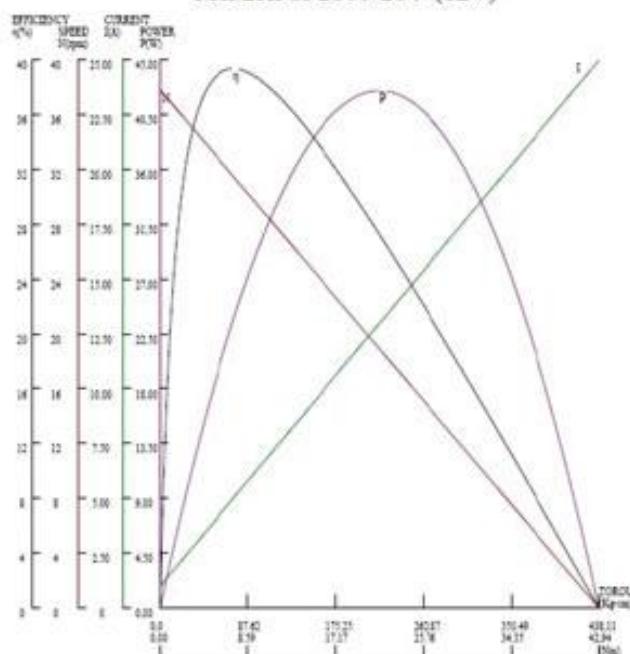


Figure 26: Hennkwell 12V DC Motor Performance Curve

b) The main purpose of this motor is to drive the lead screw. When a current is passed to drive the motor the lead screw is also driven as well because the motor is connected directly to the lead screw. So by controlling the motor rotation the lead screw is drive at the same angular velocity as the motor shaft. As it turns out the operating current is pretty low so the same motor drivers is used for the PDS and VDS.

c) PDS (same as motor) circuit diagram are provide in later section.

Motor Driver Selection

The chosen motor driver for the PDS is the same as the VDS (for more information look at the Motor Driver Selection above). This is because the current is low and this motor driver was sufficient to support the necessary current.

From Plot for the Hennwell Motor:

$$(eq. 5-5) \quad K_T = \frac{T_{MAX}}{I_{MAX}} = \frac{12 \text{ kg}\cdot\text{cm}}{5 \text{ A}} = 2.4 \frac{\text{kg}\cdot\text{cm}}{\text{A}}$$

$$(eq. 5-6) \quad I_{REQ} = \frac{T_{REQ}}{K_T} = \frac{1.407 \text{ kg}\cdot\text{cm}}{2.4 \frac{\text{kg}\cdot\text{cm}}{\text{A}}} = 0.586 \text{ A}$$

Sensors and Theory of Operation

Push Sensors

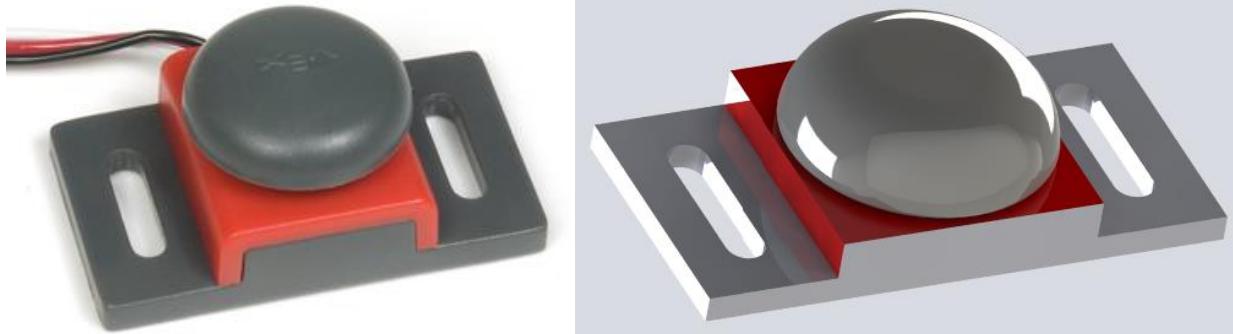


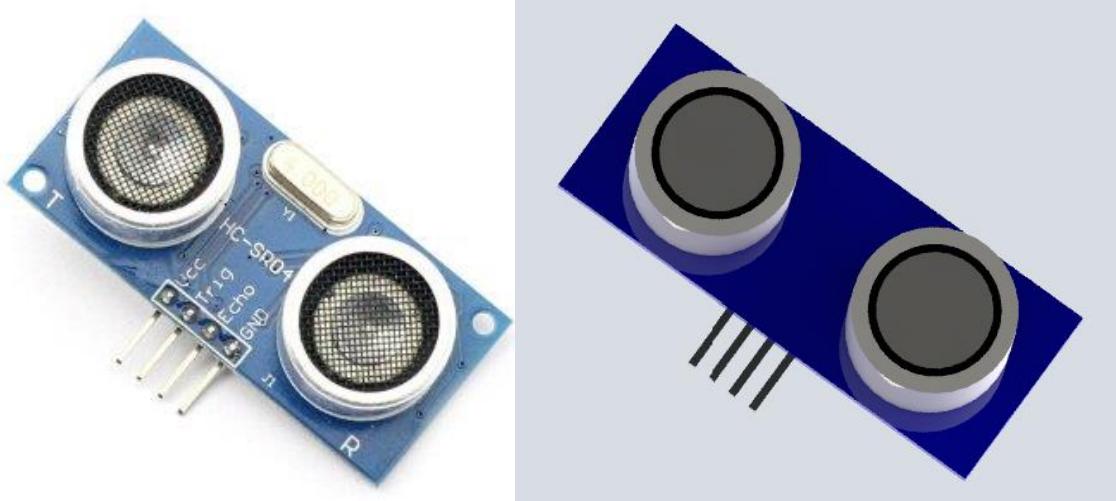
Figure 27: Vex Robotics Push Sensors (left-physical) (right-Solidworks)

- a) These Vex Robotics push sensors are best used to detect solid surface contact. This is due to the way the push sensors are designed. Depending on whether the sensors are pushed or not this create an open or close switch that controls whether current flows or not.
- b) These sensors are used to detect the ending location wall to unload and the starting location wall on its way down. To detect the wall the vehicle needs to drive into the wall and push both the buttons.
- c) Circuit diagram are provide in the circuit section.

d)

Table 23: Pros/Cons of a push sensor

Pros	Cons
Simple trigger mechanism	Require surface contact and switch compression
Easy installation	Limited operation
Mechanical mechanism	

Ultrasonic Sensor**Figure 28: Ultrasonic Module HC-SR04 (left-physical) (right-Solidworks)**

- a) These ultrasonic sensors are best used to detect solid surface. This is due to the way the sensor is design. The sensors works by sending and receiving sound waves at a target. The effect is similar to radar or sonar used to interpret an object location or distance.
- b) These sensors are used to detect the walls distance. There are two ultrasonic sensors mounted on each side used to detect the distance from each wall. By knowing the distance from each wall the vehicle can then be kept centered. The sensors are also used to help with the wall following. The sensors transmit a sound wave and receive it back. This data is converted to distance and used to calculate for possible action taken by the vehicle.
- c) Circuit diagram are provide in the circuit section.
- d)

Table 24: Pros/Cons of a ultrasonic sensor

Pros	Cons
Good precision (1mm) for our application	Surface must easily reflect sound wave
Good distance measurement (up to 1m)	Measured voltage changes when distance is below 2cm from sensor
Works for most solid surface	Depends on speed of sound
Surface and wall detection	Depend on medium sound is propagating through
Quick responses due to sound speed	Shape of surface or material (foam)

Circuit Design Description

The following is a detailed description of the robot's circuit design:

Motor Drivers Circuit

- In the motor driver circuit design we used 3 Arduino motor drivers to power 5 motors on our robot. The VDS motors are power in parallel on each side. The circuit design of the left and right motors are exactly the same (below).

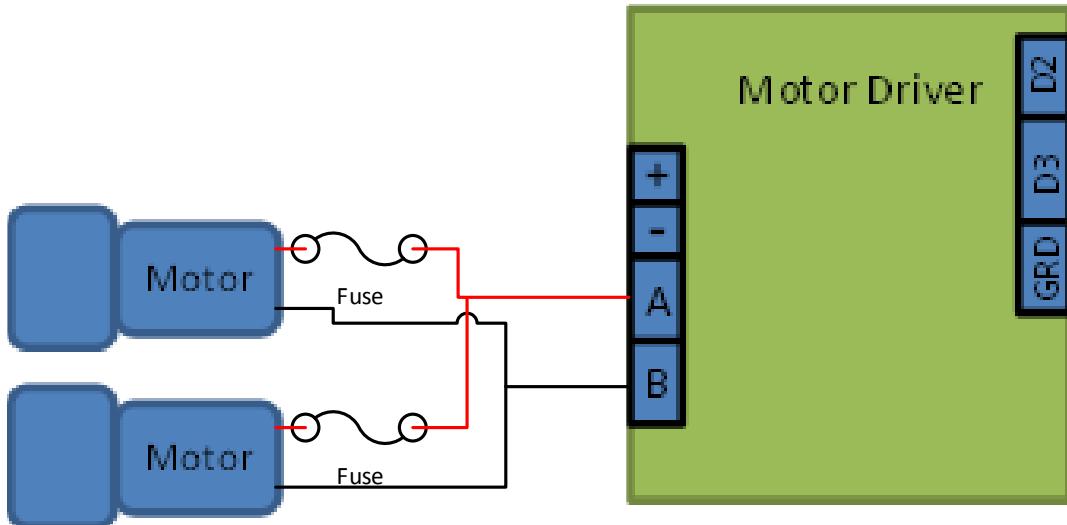


Figure 29: Motors and motor drivers circuit design for VDS (PDS has only one motor).

- There are fuses added to the input of the motor to prevent too much current from passing through the opposite motor when one of the motor is stalled.
- Fuses are also added before the motor drivers to prevent sudden current surges from frying the motor drivers. As display below the motors are all connected to the external battery supply all in parallel.
- The parallel connection of the drivers is used to provide 12V to the motors.
- The PWM signal the drivers receive from the circuit board is not directly connected to the SB-Rio to prevent possible current from surging back to the SB-Rio.
- D3 is the PWM connection. A resistor is place between the board and motor driver PWM signal to prevent possible high current from surging back to the SB-Rio. The higher the resistor's resistance the lower the current is passed through. But if the resistor is too high almost no current is passed through and therefore to weak a signal. So in this case we chose $10k\Omega$ resistors.
- All of the connections are integrated together on one breadboard hidden below the vehicle.
- The switch is added after the battery to turn on/off the motors without influencing the SB-Rio signal. This will prevent us from using the emergency stop.

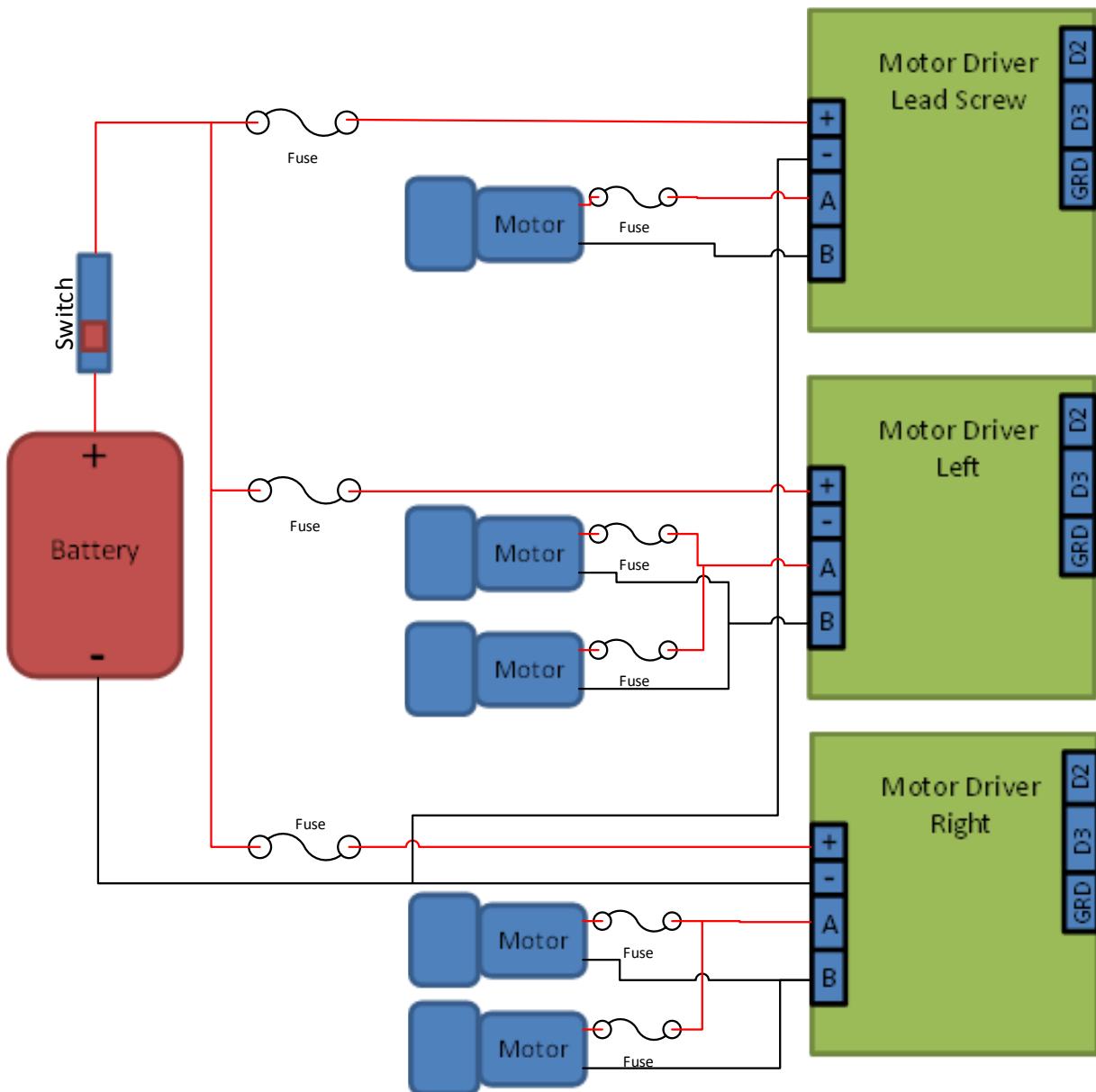


Figure 30: VDS and PDS integrated circuit design.

Sensor Circuits

- To power the sensors we needed to lower the voltage to 5V using the voltage regulator. Adding a capacitor right after the voltage regulator stabilized the voltage.
- To power the ultrasonic sensors only a parallel connection is sufficient to power it.
- For the lead screw and push button they are very similar in design. They are all basically a switch that sends a current when shorted. The lead screw switch sends a current signal when in contact to stop the motor from driving the lead screw forward.
- The switches needed resistors to prevent high current when the circuit is shorted. Higher resistors also prevent the battery from being drained too quickly.

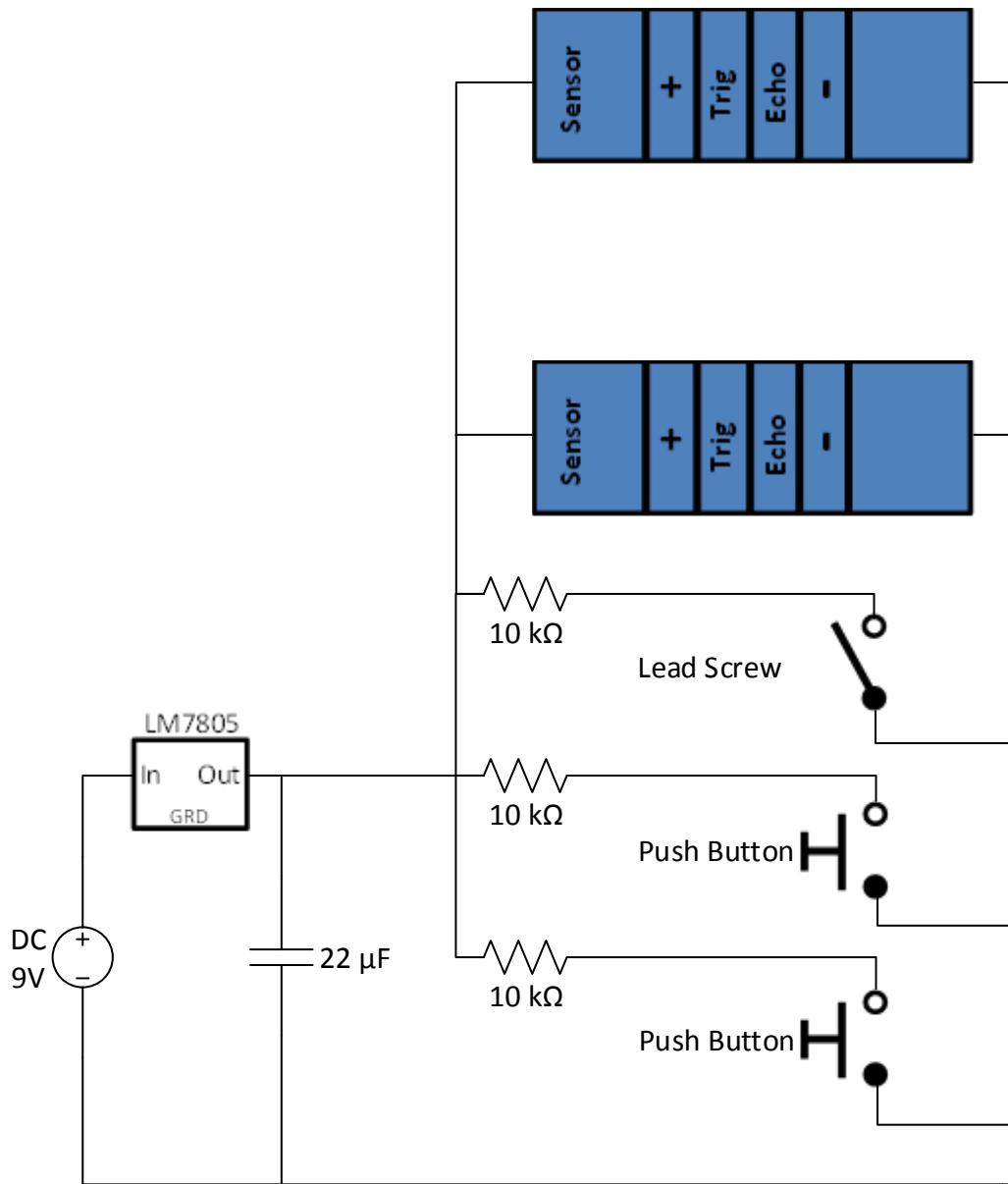


Figure 31: Sensors and PDS's switch circuit design.

Overall Circuit Wire Diagram

- The overall circuit design requires the ultrasonic sensors to be connected to the respective echo and trigger port in the SB-Rio.
- All the push button and lead screw switch are connected to digital inputs.
- The motor driver ports D3 are connected to the respective SB-Rio PWM port to provide speed signal.
- The motor driver ports D2 are connected to the respective SB-Rio digital input port to give the direction of the motor rotation.

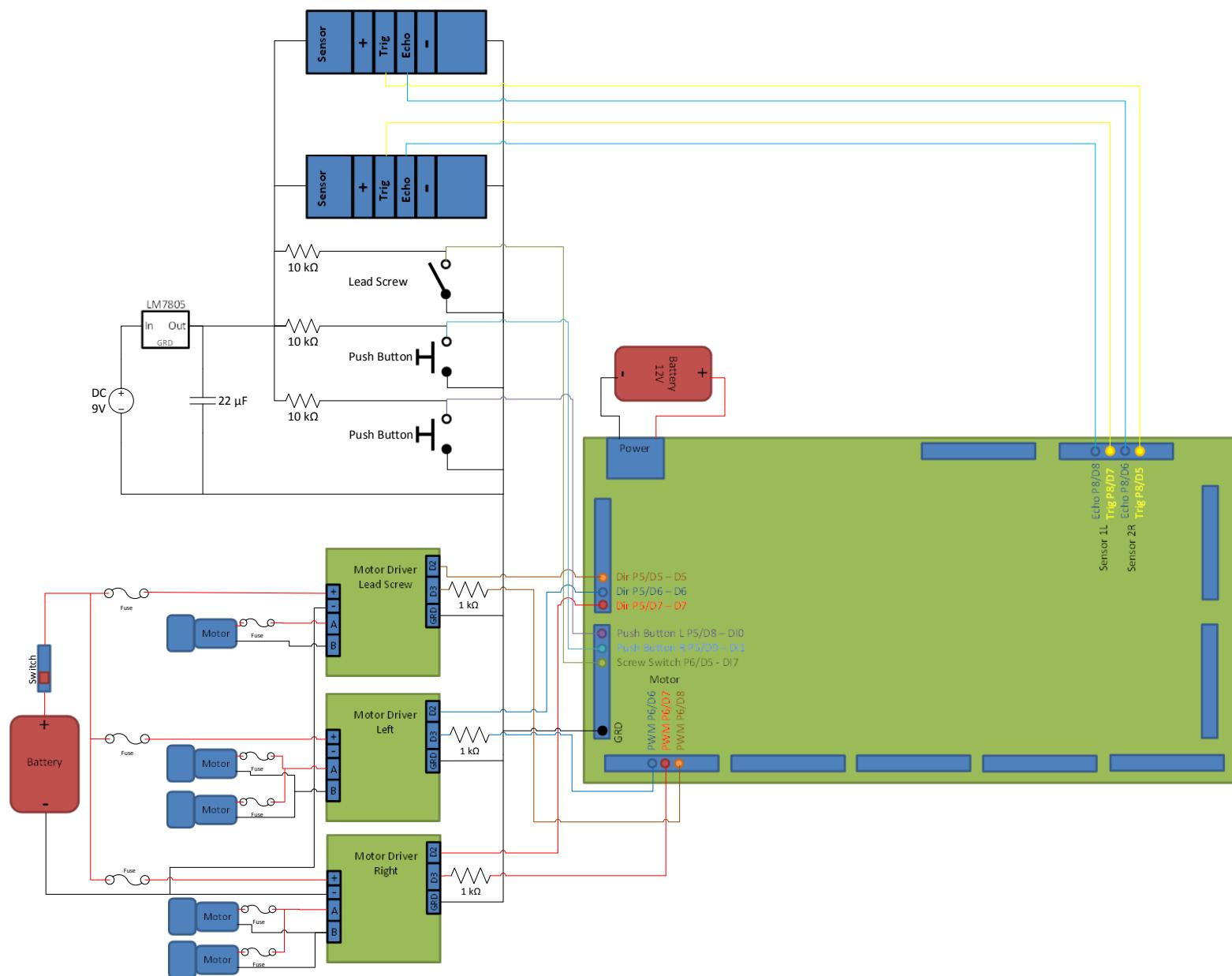


Figure 32: Complete wire diagram with VDS, PDS, and Sensor circuits.

State Diagram

Our percept sequence is based on distance measurements from two ultrasonic sensors, each mounted to a fender on the front wheel. We can then find how far off-center the vehicle is by the formula:

$$d = L - R$$

We then set a threshold, ϵ , and have our lane control activate when the vehicle is off-center enough that $d > \epsilon$.

We also implemented a spin state where if during a turn one sensor lost its signal (such as a reflection from a convex surface) the robot would drive forward, spin, then drive forward again and attempt to regain the signal.

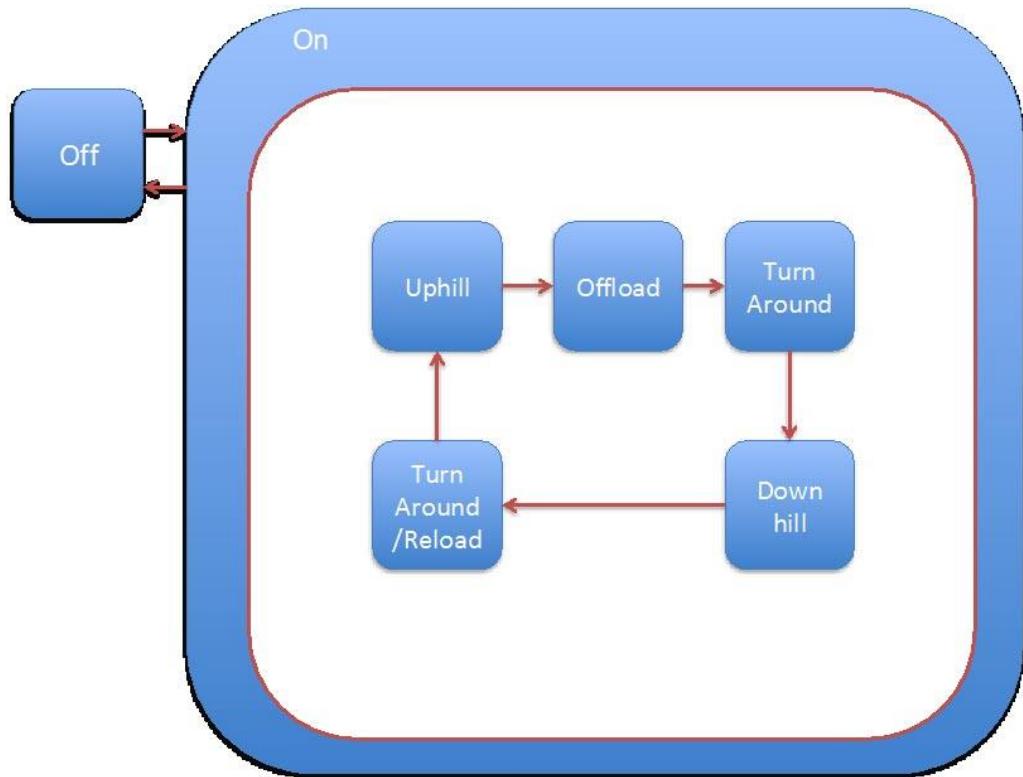


Figure 33: General Regions State Chart

Figure 33 shows the general regions of the state chart diagram. *Off* is accessible from any state to allow resetting of the delivery program. The *On* state loops through the delivery algorithm

resetting all boolean flags before repeating *Uphill*. The different *Turn Around* states differ only slightly, with the second adding an extra pause to allow reloading of disks. *Uphill* and *Downhill* are identical except for a power dampening applied to *Downhill* to reduce motor power (since the vehicle weight is much lighter).

Uphill

The default state within *Uphill* is straight and the vehicle will enter turn left or turn right once it is off-center enough to cross the threshold discussed earlier.

Once the robot has entered the *Turn Left/Right* state, the outboard and inboard motors are sent multipliers based on how far off-center it is. If the vehicle is 4cm off-center, the outboard motors would be multiplied by 2.5, if only 1.5cm off, the outboard motors would be multiplied by 1.1. This allows for more aggressive correction as the robot becomes more off track.

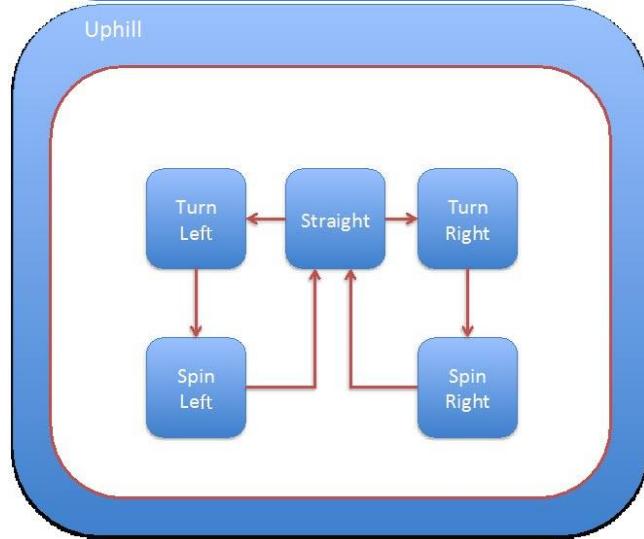


Figure 34: Uphill Statechart

Straight

Action: Left and Right motors forward at front panel setpoint.

Straight -> *Turn Left/Turn Right*

Guard: $d > \epsilon$

Action: Multiply outboard and inboard wheels by constant c_o and c_i

Turn Left/Right -> *Spin*

Guard: Outboard sensor reads less than 10cm, inboard sensor reads above 50cm

Action: Drive Forward for a count, spin for a count, drive forward for a count, return to *Straight*.

Offload

The robot enters offload when the two front buttons are depressed. If only one button is depressed, the motors on the opposite side power up to straighten the vehicle against the wall. The initial state is screw forward.

> Screw Forward

Action: Linear drive screw turns forward

Screw Forward -> *Dump*

Guard: Contacts touch, indicating delivery

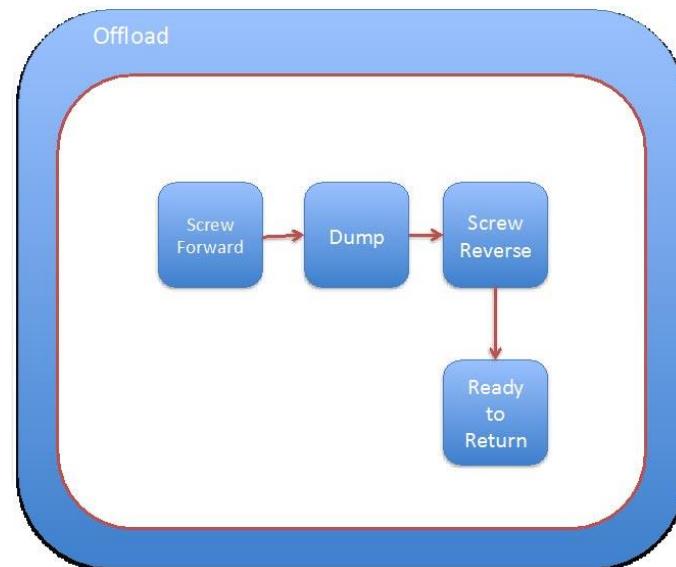


Figure 35: Offload Statechart

system in lowered position.

Action: Stop linear drive screw

Dump->Screw Reverse

Guard: Counter

Action: Linear Drive Screw turns reverse

Screw Reverse -> Ready to Return

Guard: Counter

Action: Stop linear drive screw, set boolean flag to allow entering *Turnaround*

Turnaround

Turn around backs up the vehicle with a slight right turn, then implementing a left spin. After spinning the vehicle picks up the wall and heads downhill. The boolean flags for *Ready to Return* and *Ready for Downhill* were added to prevent premature entering of *Downhill* and the *Turnaround 2/Reload*.

-> Backup

Action: Both motors reverse, left motors have a power increase.

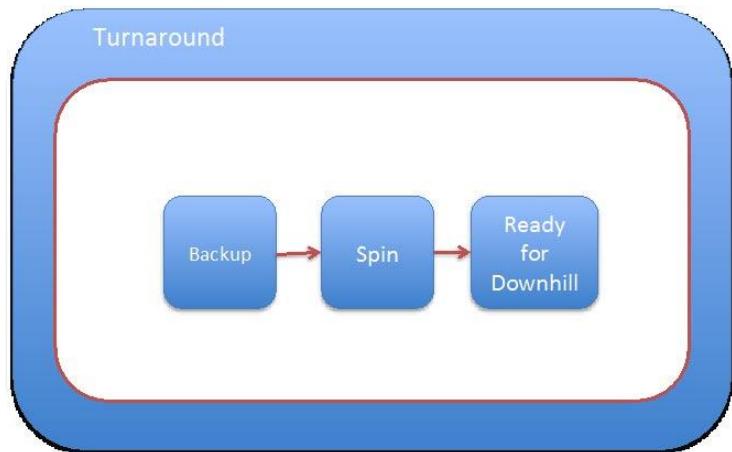


Figure 36: Turnaround Statechart

Backup->Spin

Guard: Counter

Action: Right motors forward, left motors reversed

Spin->Ready for Downhill

Guard: Counter

Action: Set boolean flag to allow entering *Downhill*.

Downhill

Downhill is then a minor variation on *Uphill* but all motor values are multiplied by a constant set at the front panel (typically 0.8)

Turnaround/Reload

Similar to *Turnaround* but with a straight *Backup* and a counted pause before returning to *Uphill*.

The advantage of using the two wall inputs is not needing a different state between the wider loading area and the main 16 inch ramp section. This would give our robot the flexibility of handling varying widths of ramps. Taking two inputs did cause more problems, where if the robot was severely oblique it would not get correct readings and would produce undesirable behavior.

6. Product Fabrication

Overview

All components of the Panhra are purchased or made in house (none are custom made by professional machinists). Due to the nature of the project, all custom parts are required to be machined by the project team in the UCLA Student/Faculty Machine Shop with traditional machining equipment and tools. For this reason, Design for Manufacturability and Assembly (DFMA) was a critical aspect of the Panhra design. At every step of detailed design, thought for easy manufacturability with traditional machines was taken into account into order to speed fabrication with minimal chances of rework necessary.

As previously mentioned, the Panhra Robot is divided into five primary subsystems. Of these subsystems, the Payload Delivery System (PDS) and Vehicle Support System (VDS) required the most custom made parts. The other subsystems primarily contained purchased parts and small mounts. The components for these systems were fabricated and assembled first as they represented the integral structural members and functional parts of the vehicle. After their fabrication, the smaller mounts and accessory parts of the other systems were manufactured. Any error in fabricating the main components could be compensated in the manufacture of the smaller components.

Vehicle Support Structure (VSS) Fabrication

The VSS establishes the chassis of the robot and is composed of 6061 aluminum L-channel extrusions and an 1/8" thick 6061 T6 aluminum plate. 6061 aluminum was chosen for its excellent strength to weight ratio and machinability. The benefit of a high strength to weight ratio is an overall weight reduction of the entire vehicle which decreases motor load and increases battery life. The L-channel extrusions serve as wheel/component mounts and stiffening members of the chassis while the plate serves as the primary mounting structure for the PDS.

Manufacturing the L-channel components required the use of a vertical bandsaw and a drill press alone, while the plate required the bandsaw, drill press, and a jigsaw. Rough edges were deburred with a belt sander and a file.

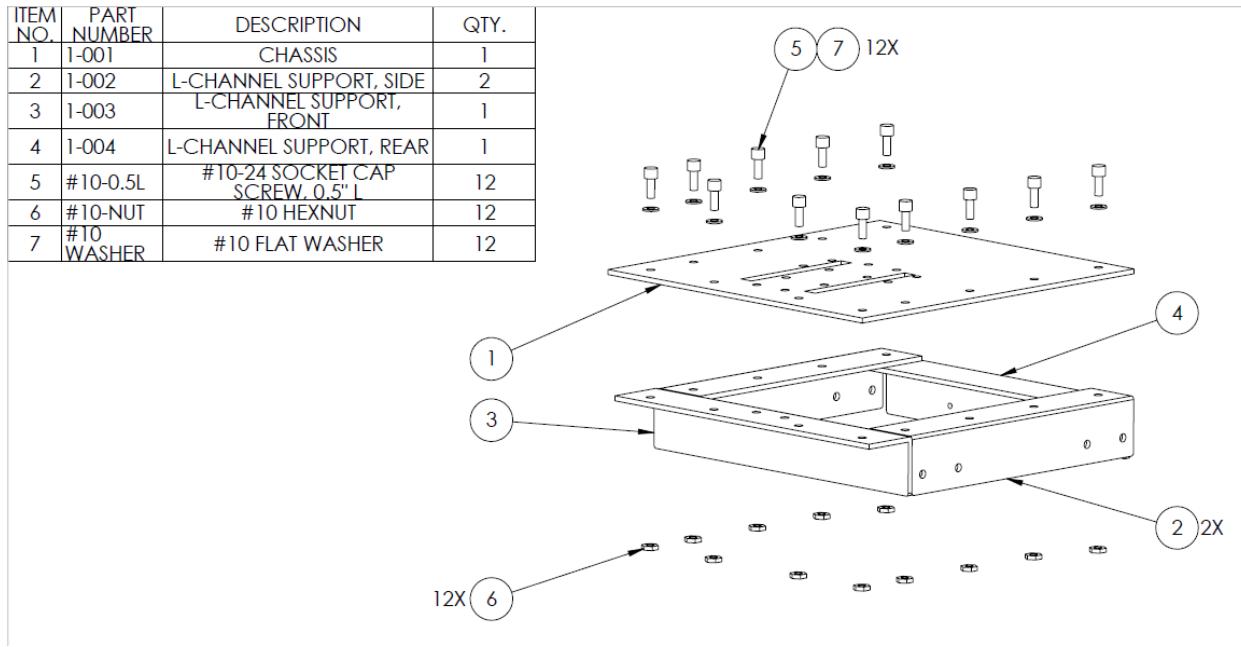


Figure 37: Vehicle Support Structure Engineering Drawing, Exploded View

Payload Delivery System (PDS) Fabrication

The PDS serves as the functional delivery system for the lead disks. Designing the PDS for manufacturability presented a difficulty as the creative nature of the system demands complexity and low tolerances. In order to perform its function, the PDS utilizes a linear actuator. The actuator requires contact between part faces with minimal normal forces between them. Care was taken to machine parts to tolerances of 0.010", and in some instances, 0.005". All components of the PDS were made from 6061 T6 aluminum for the same reasons as above, except for the spindle rod, which was made from 416 Stainless Steel for additional strength and stiffness. Most components were fabricated on the mill and drill press, while the spindle rod was fabricated on the lathe. Fabrication also involved deburring, tapping, threading, and chamfering.

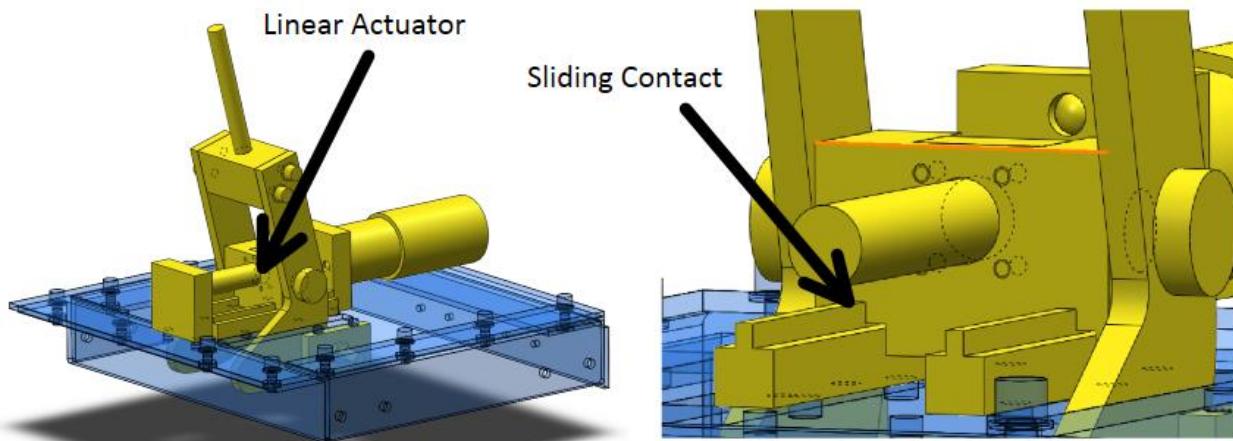


Figure 38: Vehicle Delivery System: Low Tolerance Sliding Contacts Emphasized

During manufacture of these parts, some small adjustments were made to the design of the actuator slides and drive screw bushing mounts. The team realized that the front drive screw bushing mount should not contain a through-hole, but a flat bottomed hole to constrain the drive screw from moving forward. This relieved the torque needed on the set screw holding the drive screw in place in the motor coupler. Other adjustments were small dimensional changes necessary to compensate for small dimension errors in other components.

Assembly

Lastly, assembly was taken into account in the design of the Panhra. Aside from cases where purchased components required a specific fastener, #10-32 fasteners were used throughout the construction of the Panhra in order to guard against improper fastener installation and to reduce costs (by allowing bulk purchase). Furthermore, the Panhra was designed for easy access to and removal of the SB RIO, sensors, motor drivers, breadboard, and batteries. Each of these components can be removed from the Panhra without removing any structural components. This allows for quick and simple electrical debugging as well as quick battery exchange.

The following table lists every custom made part, its material, and machines used in its manufacture:

Table 25: Parts and Machines

Subsystem	Part Description	Mill	Lathe	Drill Press	Bandsaw	Belt Sander	Other
Vehicle Drive System (VDS)							
	<i>Motor Mount</i>			x	x	x	
Payload Delivery System (PDS)							
	<i>Delivery Arm</i>			x	x	x	
	<i>Spindle</i>		x				Die
	<i>Spindle Mount</i>	x		x	x	x	Tap
	<i>Drive Nut Mount</i>	x			x	x	Tap
	<i>Actuator Slide</i>	x		x	x	x	Tap
	<i>Drive Screw Bushing Mount</i>	x			x	x	
	<i>Drive Motor Mount</i>	x			x	x	Counterbore
	<i>Drive Screw</i>	x	x		x		
	<i>Roller Mount</i>			x	x	x	Tap
Vehicle Support Structure (VSS)							
	<i>L-Channel Members</i>			x	x	x	
	<i>Chassis Plate</i>			x	x		Jigsaw
Electronics and Control System (ECS)							
	<i>Motor Driver Mount</i>			x	x	x	

7. Product Testing and Evaluation

The goal for our team from the get-go was to transport and unload three lead discs autonomously. While transporting three discs is a disadvantage competitively, our decision to use a unique unloading mechanism outweighed winning the competition. Due to the uniqueness of the unloading mechanism, carrying more than three discs seemed like a long shot.

The balance between stability and speed was at the forefront of product testing. With a full load of three discs, at one hundred percent power, the robot can probably drive in a straight line smoothly at a high speed; however, the robot cannot perform a skid steering turn at this speed. A fine line had to be drawn to decide how much speed to give up for stability. At twenty five percent motor effort, the robots steering performed great. The slow speed allowed skid steering to occur quickly enough relative to the robots speed. Unfortunately, at twenty five percent motor effort, the robot cannot drive up the fourteen degree incline with three discs loaded. After testing, a balance was found between stability and speed. Thirty eight percent motor power was the optimum output to reach our goal.

At thirty eight percent motor effort, the total runtime is seventy seconds. This is a little slower than our goal of sixty seconds, but will suffice. At forty five percent motor effort, a full run can be completed in sixty five seconds, however, the robots steering responds poorly. With seventy second runs, we expect to successfully complete four runs carrying three discs.

Four discs were tested with mixed results. The run was completed successfully at forty five percent motor effort. Driving on the ramp with the extra disc slowed the uphill portion by five seconds, as well as the unloading by one second. While the run was completed successfully, turning posed a problem. It took a lot of fine tuning to get the turning stable with three discs. With four discs, it would take countless hours to perfect the turning. The instability of four discs causes the robot to jerk when driving straight and not turn tight enough, due to the added friction on the skid turn. The benefits of using the bonus disc do not outweigh stability for us, therefore, we are comfortable using three discs, and aim to complete four solid runs.

Table 26: Breakdown of average run times

# of Discs	Uphill time (s)	Unload time (s)	Downhill time (s)	reload time (s)
3	35	5	30	5
4	40	6	30	6

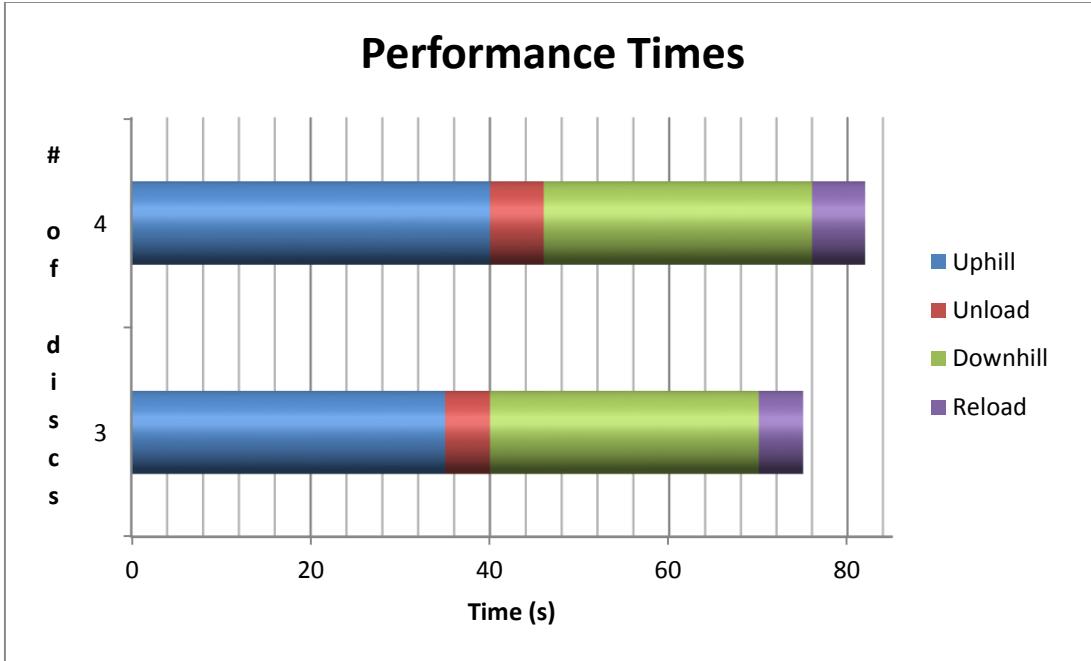


Figure 39: Performance Times

The increased time while carrying four discs is due to the heavier load. The motor power can be increased to compensate for the heavier, however, steering is not sufficient. Unloading increases by a second due to the fact the lead screw motor is slowed by the heavier load, this is not a negative impact. Reload time also adds a second, it takes about a second a disc, with a two second safety window based on averages during testing.

Unloading time can be sped up by increasing the lead screw's motor effort. If the discs are unloaded too fast, a risk of tipping arises from the impulsive torque, we do not want to risk this. From testing, sixty percent unloading power delivered the load smoothly in 5 seconds.

8. Work Breakdown Schedule

Work Breakdown Schedule Dictionary

Table 27: Work Breakdown Schedule Dictionary

WBS Element	Description
Panthra Transporter	Project title, main deliverable (transporter)
1. Design 1.1 Mechanical Systems 1.1.1 Analysis 1.2 Electrical System 1.2.1 Analysis	Design of transporter, complete CAD model Drive and delivery systems Force, torque, FEM analysis Includes SB rio board, sensors, etc. Circuit analysis
2. Material Procurement 2.1 Bill of Materials 2.2 Prices 2.3 Purchasing	Self explanatory Get from CAD model Research best prices Purchase all materials
3. Manufacturing 3.1 Create drawings 3.2 Machining 3.2.1 Distribute work	Self explanatory Create using CAD model Machine all metal parts Split up work load
4. Assembly of Mechanical Parts	Self explanatory
5. Test Electrical Components 5.1 SB Rio Board 5.2 Sensors 5.3 Motor Controllers	Self explanatory Test battery, board, connectors Test connectors Test connectors
6. Circuit Diagrams 6.1 Data sheets 6.2 Draw circuit diagrams by hand 6.3 Draw circuit diagrams on computer	Self explanatory Look up data sheets for electrical components Self explanatory Self explanatory
7. Assemble Electrical Components 7.1 Mount components on transporter 7.2 Route all wires using circuit diagrams 7.3 Clean up/organize wires	Self explanatory Self explanatory Self explanatory Clean up wires using zip ties or other method
8. Programming	Program transporter to move, deliver load autonomously
9. Testing/Fine Tuning 9.1 Test mechanical parts 9.2 Test and fine turn programming	Self explanatory Test mechanical parts, replace broke parts Fine tune programming for best results
10. Demo/Competition	Successfully deliver load and return to starting point

Gantt Chart

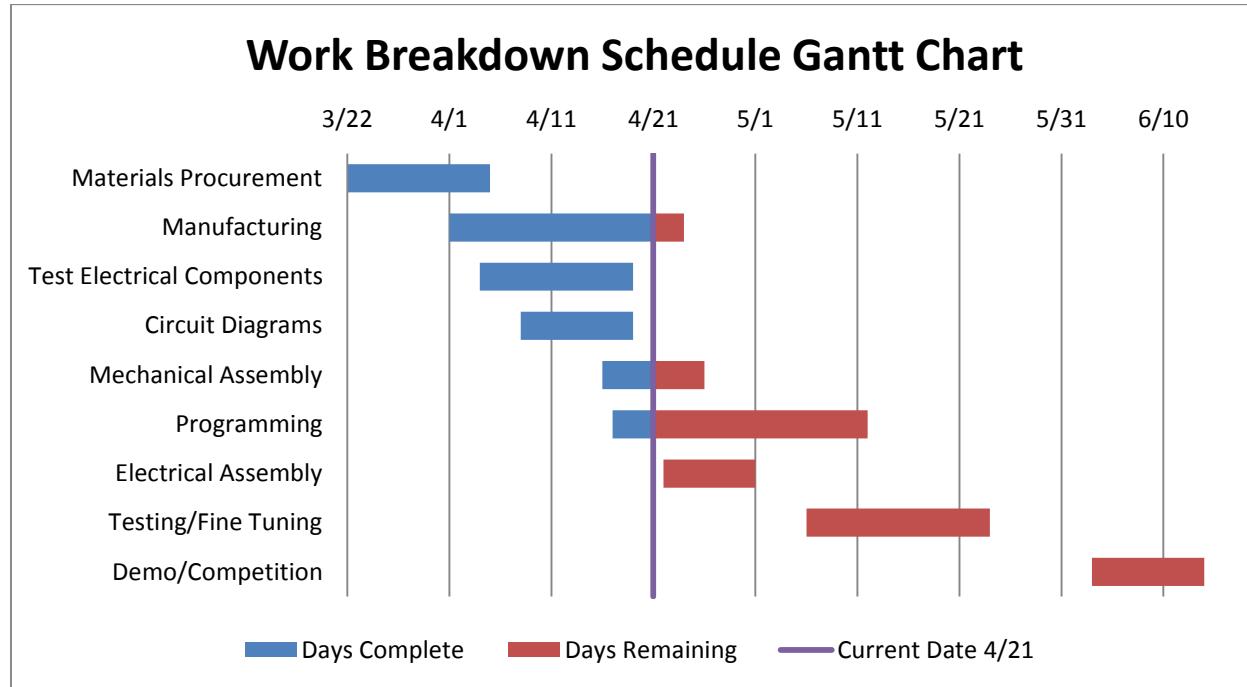


Figure 40: Schedule Gantt chart. See work breakdown schedule dictionary for more detail on these milestones

The Gantt Chart provides a good portrayal of our location throughout the project. This was mostly due to the fact that most things were done consistently each day. The group members tasks were distributed in an organized manner such that everyone always have certain tasks to accomplish. Throughout the whole process the steps that makes this possible is the determination of the individual group member that made this possible.

9. Bill of Materials and Procurement

Our CAD model has been broken up into the system subassemblies:

- 1- Vehicle Support Structure
- 2- Electronics and Control System
- 3- Vehicle Drive System
- 4- Payload Delivery System
- 5- Vehicle Sensor Array

The part numbers are broken up and numbered according to these subassemblies.

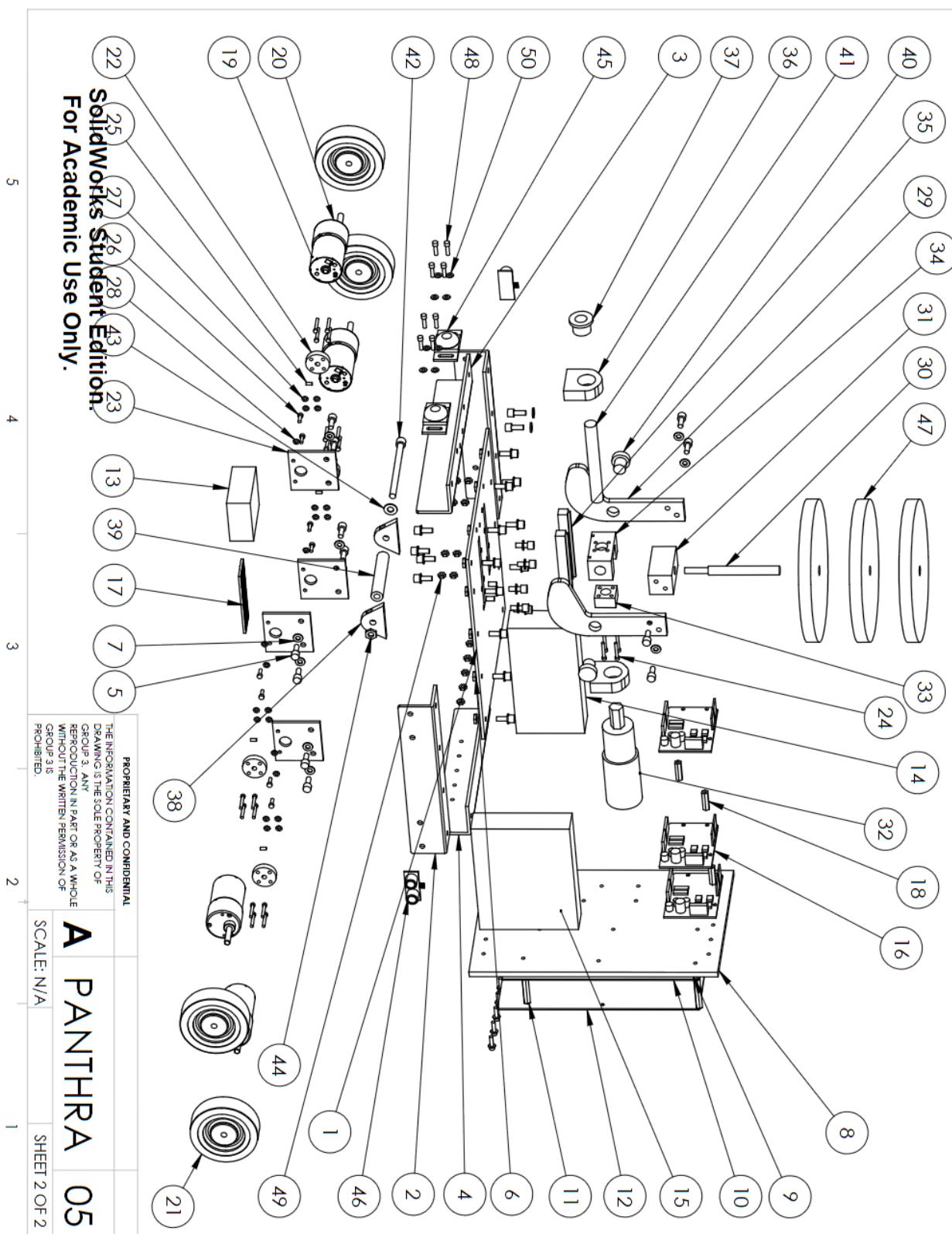
ITEM NO.	PART NUMBER	DESCRIPTION	QTY.
1	1-001	CHASSIS	1
2	1-002	L-CHANNEL SUPPORT, SIDE	2
3	1-003	L-CHANNEL SUPPORT, FRONT	1
4	1-004	L-CHANNEL SUPPORT, REAR	1
5	#10-0.5L	#10-24 SOCKET CAP SCREW, 0.5" L	36
6	#10-NUT	#10 HEXNUT	12
7	#10 WASHER	#10 FLAT WASHER	36
8	2-001	SB RIO BASE PLATE	1
9	2-002	HEX STANDOFF, .48"	4
10	2-003	SB RIO BOARD	1
11	2-004	HEX STANDOFF, 1.48"	4
12	2-005	SB RIO ADD-ON BOARD	1
13	2-006	SB RIO BATTERY	1
14	2-007	BATTERY PACK, (4) D SIZE	1
15	2-008	BATTERY PACK, (6) D SIZE	1
16	2-009	CYTRON DC MOTOR CONTROLLER	3
17	2-010	BREADBOARD	1
18	#4-STANDOFF	HEX STANDOFF	6
19	3-001	TETRIX DC MOTOR BASE	4
20	3-002	TETRIX DC MOTOR HEAD	4
21	3-003	BANEBOOTS 2.875" WHEEL	4
22	3-004	AXLE SET COLLAR	4
23	3-005	MOTOR MOUNT PLATE	4
24	#4-0.75L	#4 SOCKET CAP SCREW	20
25	#4ss-0.25L	#4 SET SCREW	4
26	M3x0.5-0.375L	M3X0.5 SOCKET CAP SCREW	8

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DRAWING IS THE SOLE PROPERTY OF
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UNAUTHORIZED REPRODUCTION IN PART OR AS A WHOLE
WITHOUT THE WRITTEN PERMISSION OF
GROUP 3 IS
PROHIBITED.

ITEM NO.	PART NUMBER	DESCRIPTION	QTY.
27	#4 WASHER	#4 FLAT WASHER	16
28	M3 WASHER	M3 WASHER	8
29	4-010	PAYLOAD CARRY ARM	2
30	4-011	PAYLOAD SPINDLE	1
31	4-012	SPINDLE MOUNT	1
32	4-102	HENK WELL DC MOTOR ASSEMBLY	1
33	4-005		1
34	4-009		1
35	4-001	LINEAR ACTUATOR SLIDE	2
36	4-002	DRIVE SCREW BEARING MOUNT	2
37	4-003	DRIVE SCREW BEARING	1
38	4-004	ROLLER MOUNT	2
39	4-006	ROLLER	1
40	4-007	0.5" SHOULDER SCREW	2
41	4-008	0.5" LEAD SCREW, 4" L	1
42	.25-3.0L	0.25" SOCKET CAP SCREW	1
43	.25 WASHER	0.25" WASHER	1
44	0.25-NUT	0.25" HEX NUT	1
45	5-001	VEX BUMPER SENSOR	2
46	5-002	ULTRASONIC SENSOR	2
47	Lead Disk weight 5.75lb	LEAD DISC, 5.75LB, 6" DIAMETER	3
48	#6-.5L	#6-32 SOCKET CAP SCREW	13
49	#6-NUT	#6 HEX NUT	13
50	#6 WASHER	#6 FLAT WASHER	13

UNLESS OTHERWISE SPECIFIED:	DRAWN	NAME	DATE
DIMENSIONS ARE IN INCHES TOLERANCES ARE FRACTIONAL: ANGULAR MACH. BEND :	CUNDERHILL	03/20/2013	
TWO PLACE DECIMAL : THREE PLACE DECIMAL :	CHECKED	ENG APPR.	
INTERPRET GEOMETRIC TOLERANCING PER:	ENG APPR.	MFG APPR.	
MATERIAL	N/A	Q.A.	
FINISH	N/A	COMMENTS:	
DO NOT SCALE DRAWING		SIZE	DWG. NO.
		A	PANTHRA
		REV	05
SCALE: N/A WEIGHT: N/A		SHEET 1 OF 2	



We bought all of the parts ourselves. We have done a lot of research on vendors and believe our parts are reasonably prices. Some of these parts will be ordered online and some will be purchased in person. We came in just under the \$350 budget.

10. Design Requirement Satisfaction

Subsystem	Requirements	Result
VDS	Sufficient motor output to propel vehicle up ramp and through course with desired speed and a 3 disk payload.	The motor calculations in Part 3 show that the 4 Tetrix motors provide ample power to drive the vehicle under load with a run time of approximately 40 seconds.
PDS	Strong enough material not to deform under requisite load. Lower load quasi-statically to offload position and self righting of spindle.	Calculations in Part 6 show that a spindle made of High Strength ASTM-A242 Steel should not deform under the load of 3 disks The curved design of <i>the Fang</i> will lower the arm quasi-statically and return the arm to the upright position through use of only a lead screw drive system.
	Sufficient motor output to advance arm.	The motor calculations in Part 6 show that the Henkwell 12V motor provides ample power to advance the PDS under load.
VSS	Sufficient strength to hold on subassemblies in within the requisite size constraints and a center of gravity (CG) somewhat centered and the ability to remain upright in the offload position.	The vehicle falls within the requisite dimensional constraints with a CG within the wheelbase. In the offload position, the vehicle will utilize a lip on the front to rest on the maze wall and prevent flipping. Since we have not yet learned the FEA in Solidworks, our strength analysis in this area is incomplete until next quarter. Using common sense and machining experience, our structure should be strong enough to handle all necessary loads.
ECS	Must fit within dimensional constraints.	The current configuration of the vehicle fits within the requisite dimensional constraints.

	Provide requisite current and stored power to handle multiple runs	By selecting D batteries for the final run, the ECS should provide enough amp-hours to conduct several runs.
VSA	Ultrasonic sensors must have minimum range to detect walls at close proximity	The ultrasonic sensors have a minimum range of 2cm. Even with that case, by placing two sensors on each side let us read distance from both wall. This will always give us good results from one sensor and provide smooth turns.

11. Conclusion

From our original three designs we have narrowed it down to a combination of the *Panthro* and *Cheetara* base on the analysis we did with the conceptual tree. Our conceptual design from these two combinations creates the *Panthra*. This robot has met the requirements for our needs based on our initial calculations. *Panthra* is distinct from the other two designs because it incorporates the innovative spindle arm with the four wheel drive system for maneuverability. The spindle arm is a plus, because we wanted a design that uses the force of gravity to help drop the disks and be able to extend and retract to move our center of gravity only when we are unloading. As for the four wheel base, we chose this mostly for maneuverability. From our knowledge and objective tree we avoided the track system since it has more friction when turning when compared to the four wheel vehicle turning. From this analysis we have *Panthra*.

During the manufacturing process of *Panthra* we smoothly create the necessary parts for the VSS and PDS without many problems. We also perform a FEM analysis and find out that the stress was an order of magnitude below the yielding stress making the PDS safe enough to operate without failure. Following the Gantt chart schedule we finished the base as planned assemble the circuits and begin the programming. The schedule where exactly followed due to the fact that the group was extremely organized and workload were distributed evenly.

The lessons we mostly learn where picked up during the programming and organization. These lesson for programming where techniques to improve the smoothness in wall following. For organization we learn to keep work distributed and time managed. We started the programming early and worked on it consistently giving us enough time to have a tune up.

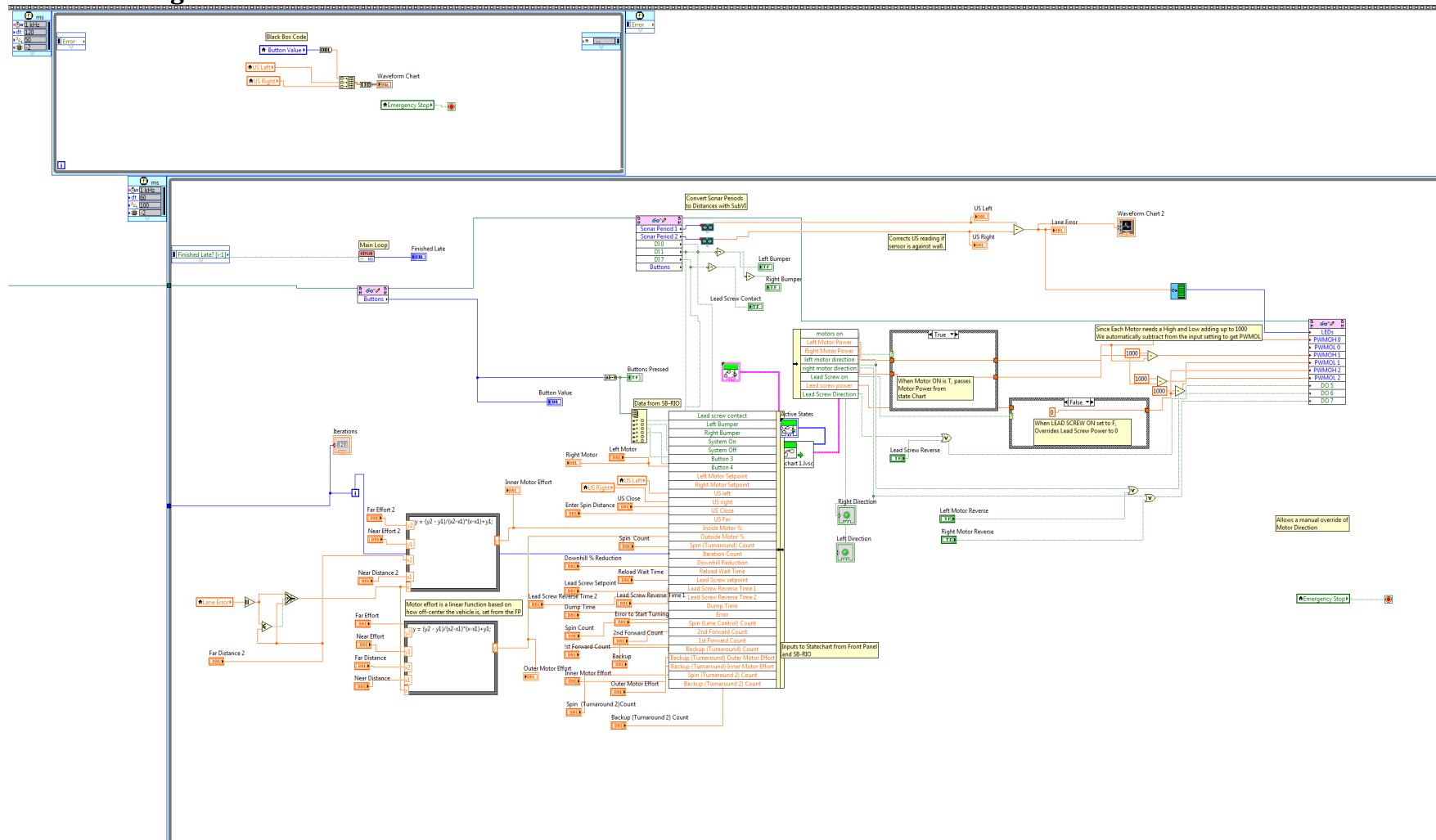
With this our successful “Panthra Project” has finally come to an end accomplishing the high level requirements we set out to perform. Overall, *Panthra* is a huge success completing the run smoothly as we desire.

12. Reference

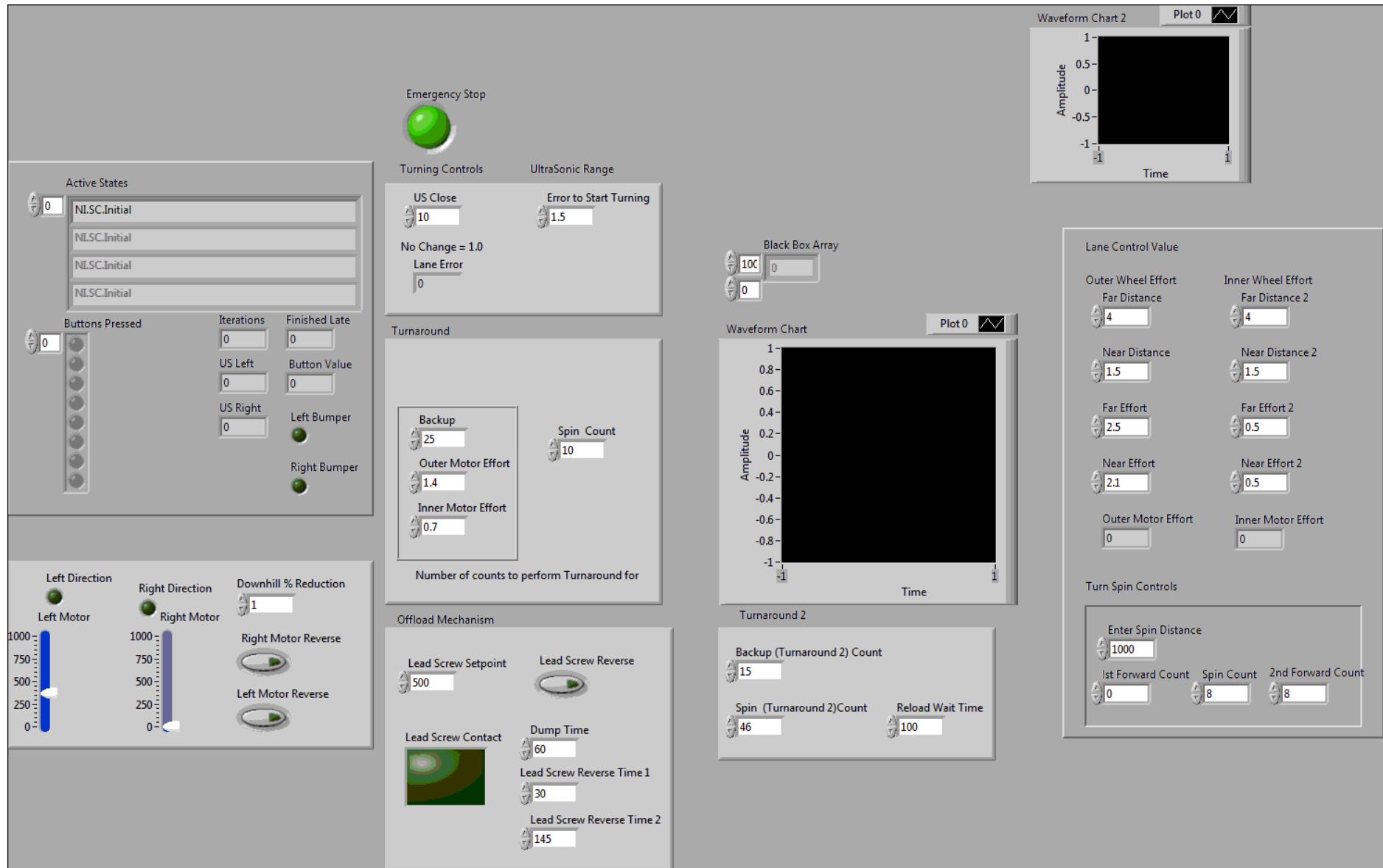
- [1] Ugural, A. *Advanced Mechanics of Materials and Applied Elasticity* (5th ed.). Prentice Hall 2012.
- [2] *BaneBots Wheel*. (2013, March 1). Retrieved from <http://banebots.com>
- [3] *General Specifications for Tetrix DC Drive Motor*. (2011, January 1). Retrieved February 5, 2013
- [4] International Atomic Energy Agency (IAEA). (2006). *The Safe Transport of Radioactive Materials 4th Edition*.
- [5] Klesov, A. A. (2007). *Wood-plastic Composites*. Hoboken, NJ: Wiley-Interscience.
- [6] Schaefer, R. (2013, Winter). Mechanical and Aerospace Engineering 162 Ramp Detail.
- [7] Schaefer, R. (2013, Winter). Mechanical And Aerospace Engineering 162D Lecture 2 Mechanical Engineering Based Design Approach to the Transporter.
- [8] Schaefer, R. (2013, Winter). Mechanical And Aerospace Engineering 162D Lecture 5A – Smart Motor Sizing.
- [9] Rolling Friction and Rolling Resistance. (n.d.). *The Engineering Toolbox* . Retrieved March 1, 2013, from http://www.engineeringtoolbox.com/rolling-friction-resistance-d_1303.html

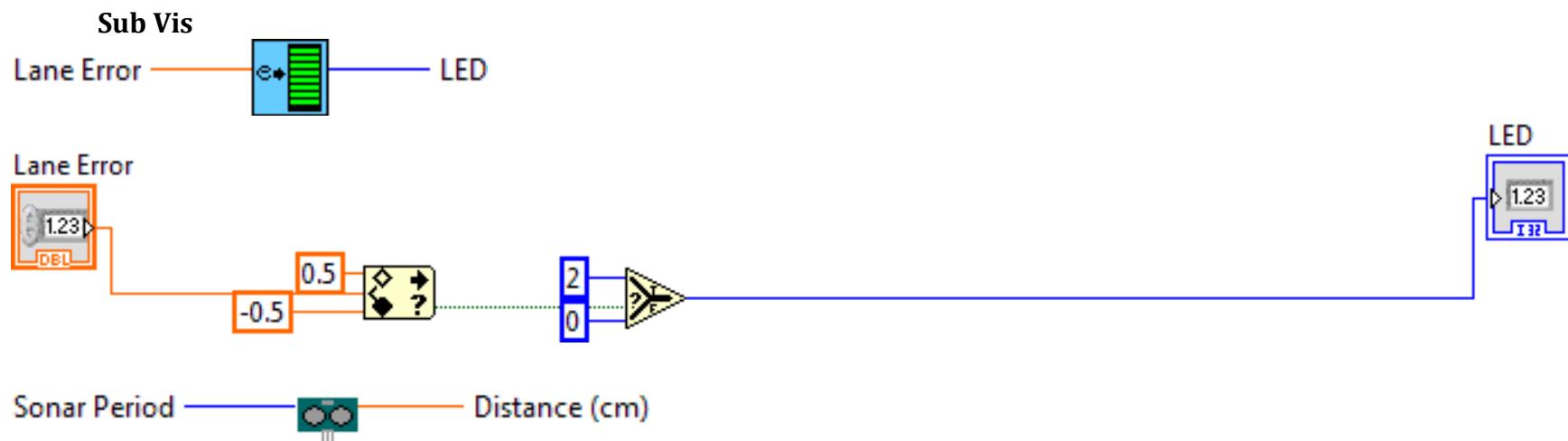
13. Appendix

Labview Code Block Diagram

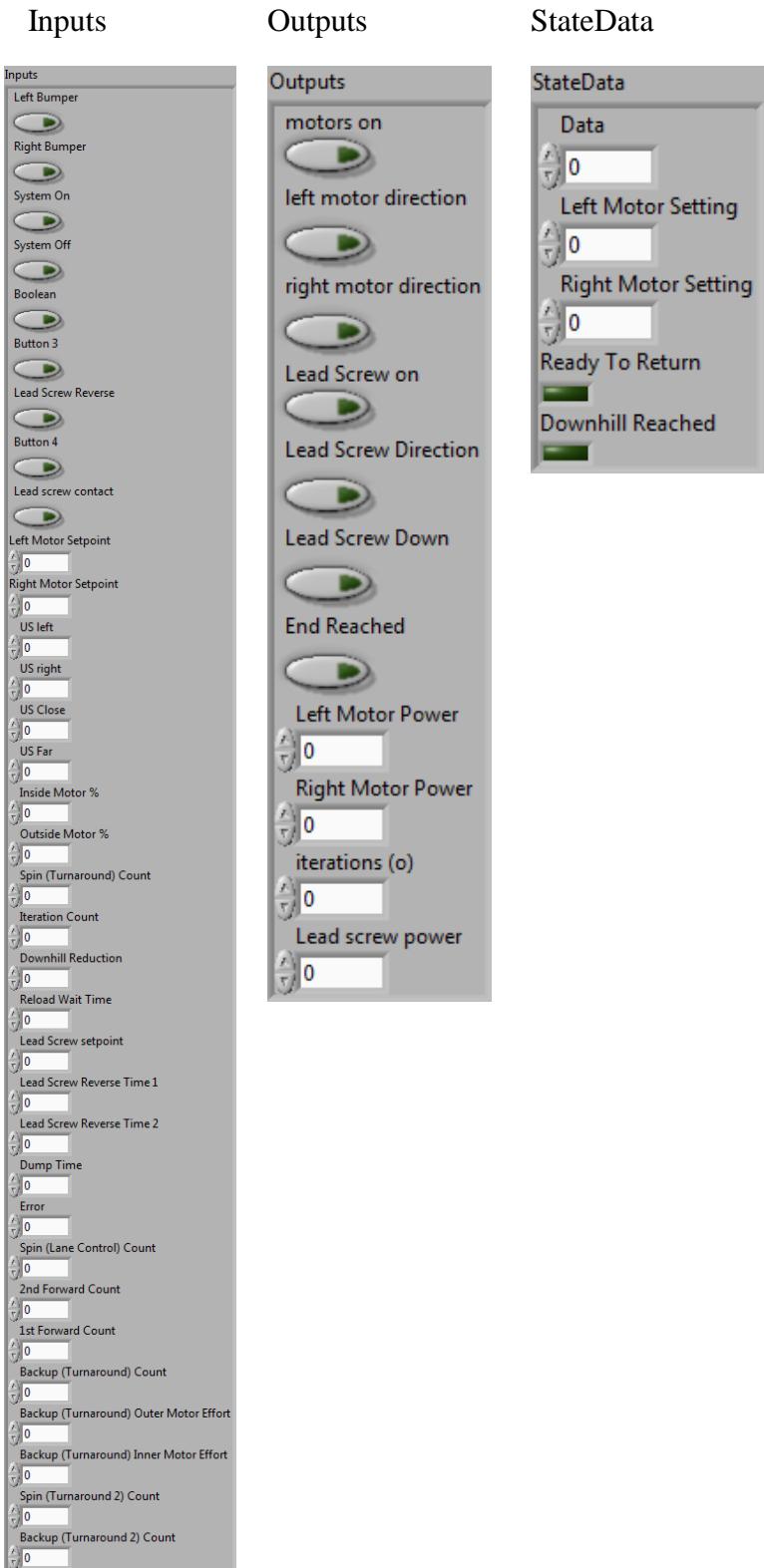


Front Panel





State Chart

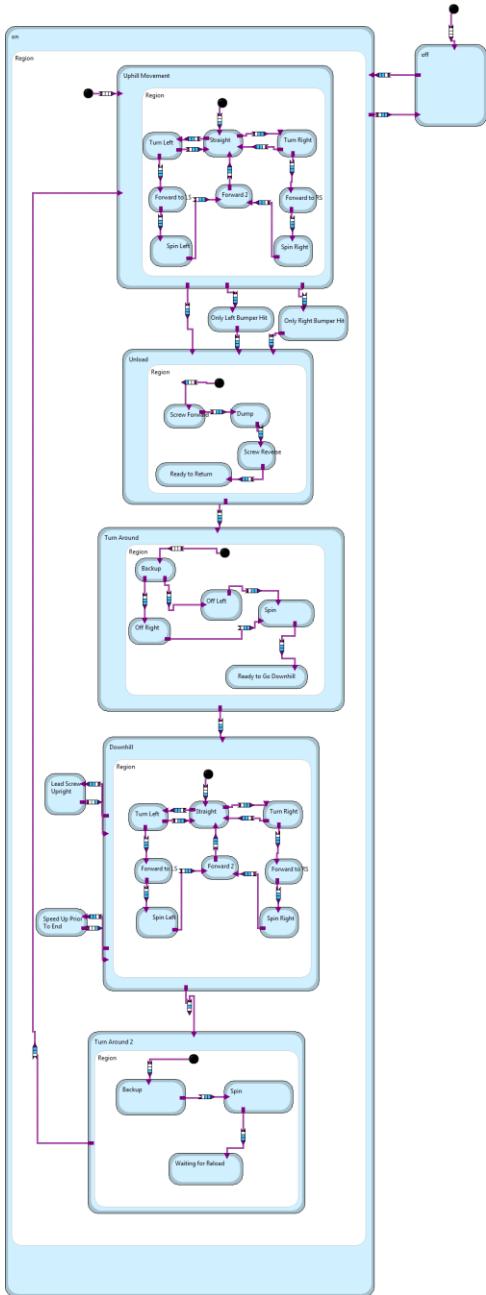


Code Generation Properties

Execution Target	Real Time
Usage	Synchronous
Queue Settings	Not Applicable
Internal trigger queue size	10
Debugging	Disabled
Reset output values after each iteration	FALSE
Guard or Action VI reentrancy	Preallocate clone for each instance

Statechart Diagram

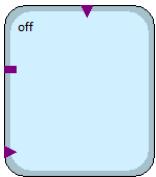
Main Diagram



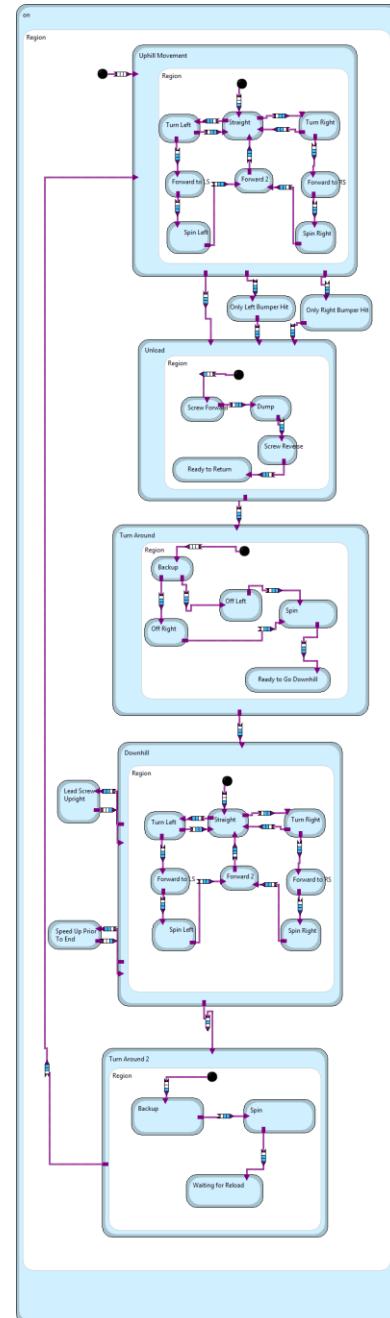
Contained Objects	Initial , off , on
Contained Transitions	Transition , Transition , Transition

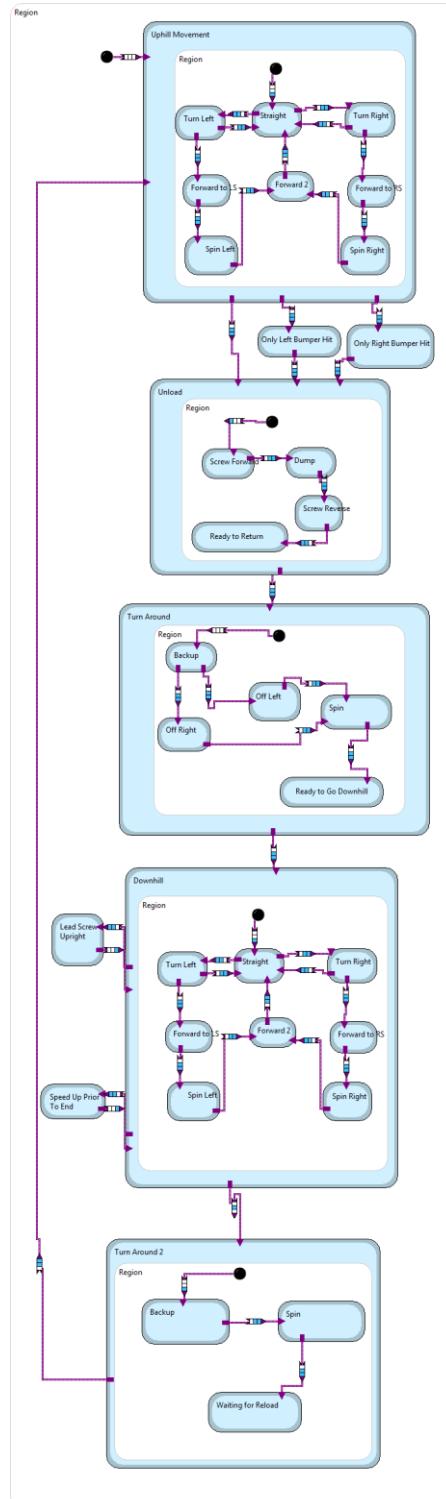
A Initial

Description	-
Destination	Transition ->off
Containing Region	Main Diagram

B off

Description	-
Source	Initial ->Transition , on ->Transition
Destination	Transition ->on
Containing Region	Main Diagram

C on



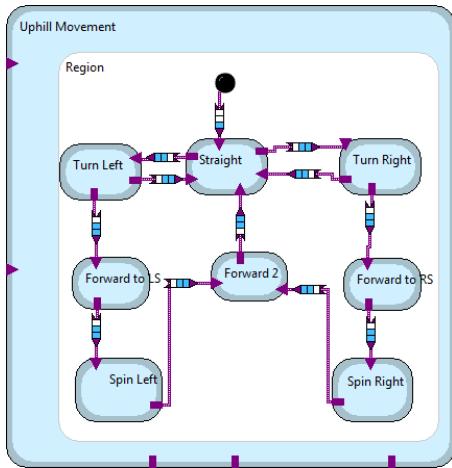
Description	-
Source	off -> Transition
Destination	Transition ->off
Containing Region	Main Diagram
Contained Regions	Region

C.1 Region

Description	-
Containing State	on
Contained Objects	Uphill Movement , Unload , Turn Around , Turn Around 2 , Initial , Downhill , Lead Screw Upright , Speed Up Prior To End , Only Left Bumper Hit , Only Right Bumper Hit
Contained Transitions	Transition , Transition

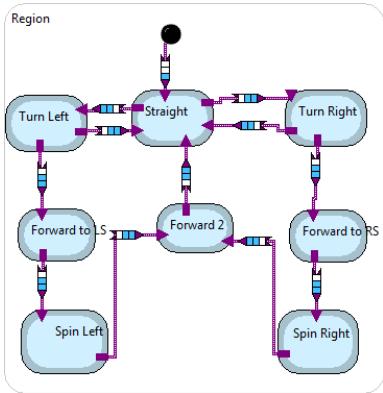
C.1.A Initial

Description	-
Destination	Transition -> Uphill Movement
Containing Region	Region

C.1.B Uphill Movement

Description	-
Source	Initial -> Transition , Turn Around 2 -> Transition
Destination	Transition -> Unload , Transition -> Only Left Bumper Hit , Transition -> Only Right Bumper Hit
Containing Region	Region
Contained Regions	Region

C.1.B.1 Region



Description	-
Containing State	Uphill Movement
Contained Objects	Straight , Turn Right , Turn Left , Initial , Spin Left , Spin Right , Forward 2 , Forward to LS , Forward to RS
Contained Transitions	Transition , Transition

C.1.B.1.A Initial

Description	-
Destination	Transition ->Straight
Containing Region	Region

C.1.B.1.B Straight

Description	-
Source	Initial ->Transition , Turn Left ->Transition , Turn Right ->Transition , Forward 2 ->Transition
Destination	Transition ->Turn Left , Transition ->Turn Right
Containing Region	Region

C.1.B.1.C Turn Right

Description	-
Source	Straight ->Transition
Destination	Transition ->Straight ,
Containing Region	Region

C.1.B.1.D Turn Left



Description	-
Source	Straight ->Transition
Destination	Transition ->Straight
Containing Region	Region

C.1.B.1.E Spin Left



Description	-
Source	Forward to LS ->Transition
Destination	Transition ->Forward 2
Containing Region	Region

C.1.B.1.F Spin Right



Description	-
Source	Forward to RS ->Transition
Destination	Transition ->Forward 2
Containing Region	Region

C.1.B.1.G Forward 2



Description	-
Source	Spin Left ->Transition, Spin Right ->Transition
Destination	Transition ->Straight
Containing Region	Region

C.1.B.1.H Forward to LS



Description	-
Source	Turn Left ->Transition
Destination	Transition ->Spin Left
Containing Region	Region

C.1.B.1.I Forward to RS



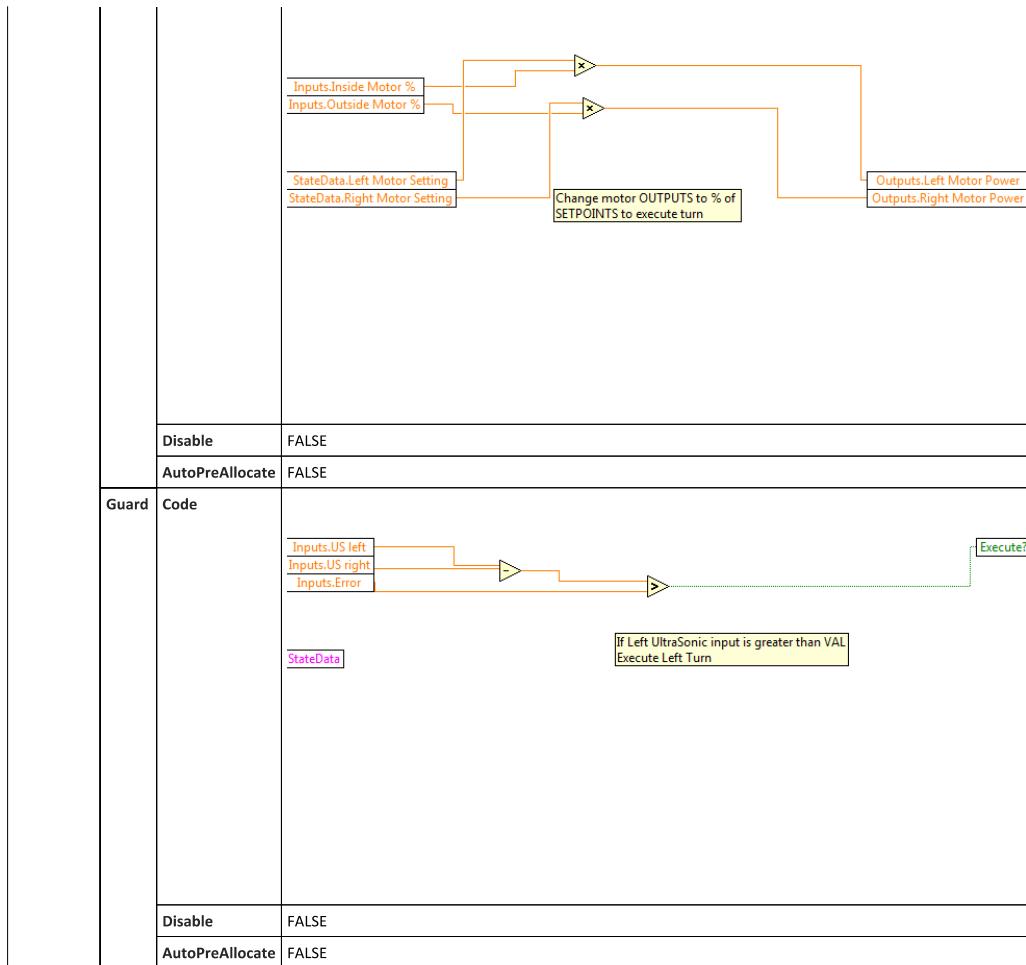
Description	-
Source	Turn Right ->Transition
Destination	Transition ->Spin Right
Containing Region	Region

C.1.B.1.J Transition

Description		-
Source		Initial
Destination		Straight
Containing Region		Region
Transition Code	Description	
	Trigger List	{Responds to all triggers}
	Action	Code
		<pre> graph LR A[Inputs.Left Motor Setpoint] --> J(()) B[Inputs.Right Motor Setpoint] --> J C[StateData.Left Motor Setting] --> J J --> D[Outputs.motors on] J --> E[Outputs.Left Motor Power] J --> F[Outputs.Right Motor Power] J --> G[Outputs.left motor direction] J --> H[Outputs.right motor direction] J --> I[Outputs.End Reached] </pre>
	Disable	FALSE
	AutoPreAllocate	FALSE

C.1.B.1.K Transition

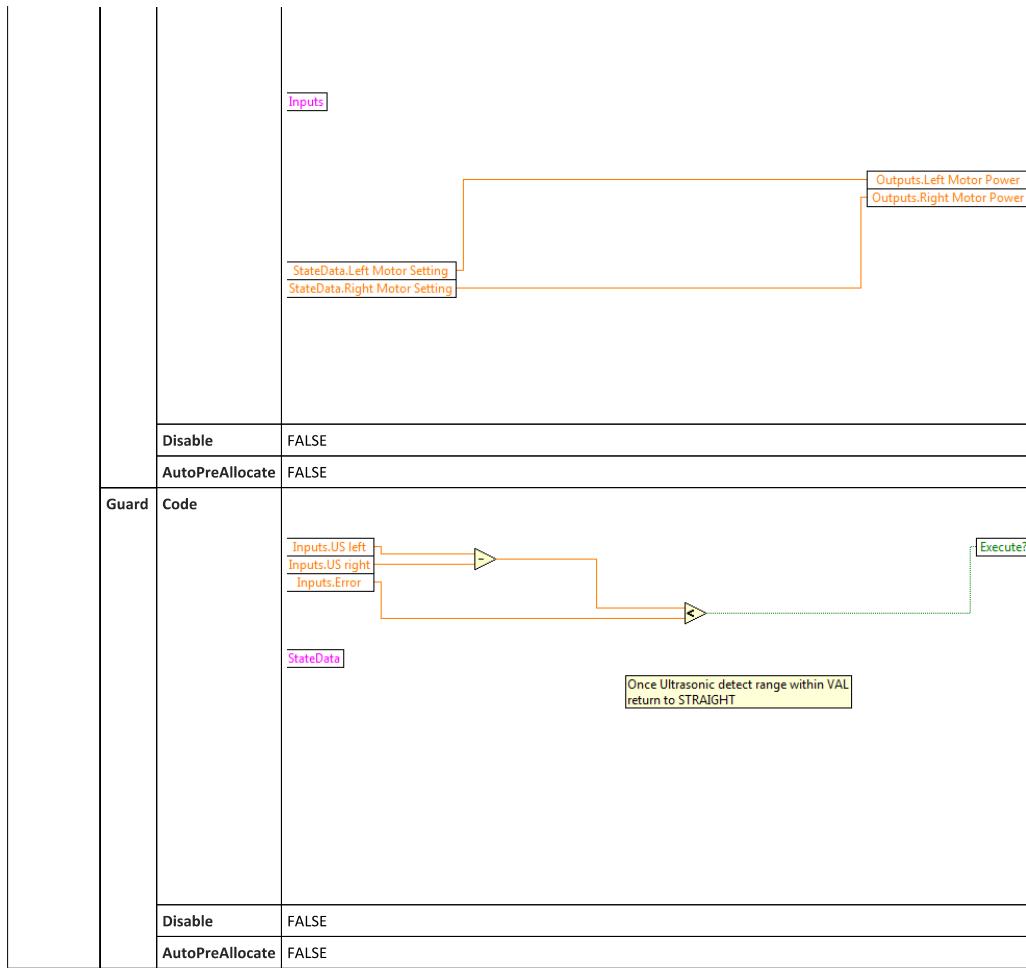
Description		-
Source		Straight
Destination		Turn Left
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code



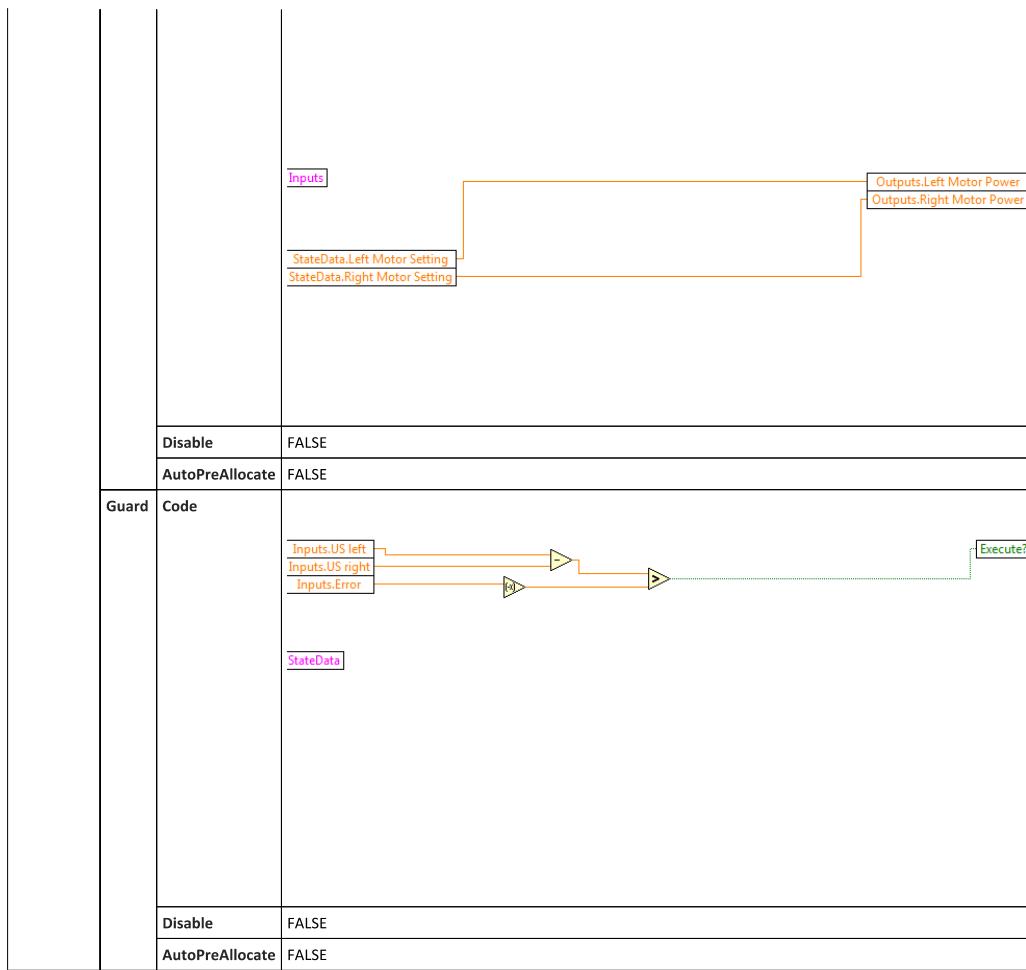
C.1.B.1.L Transition



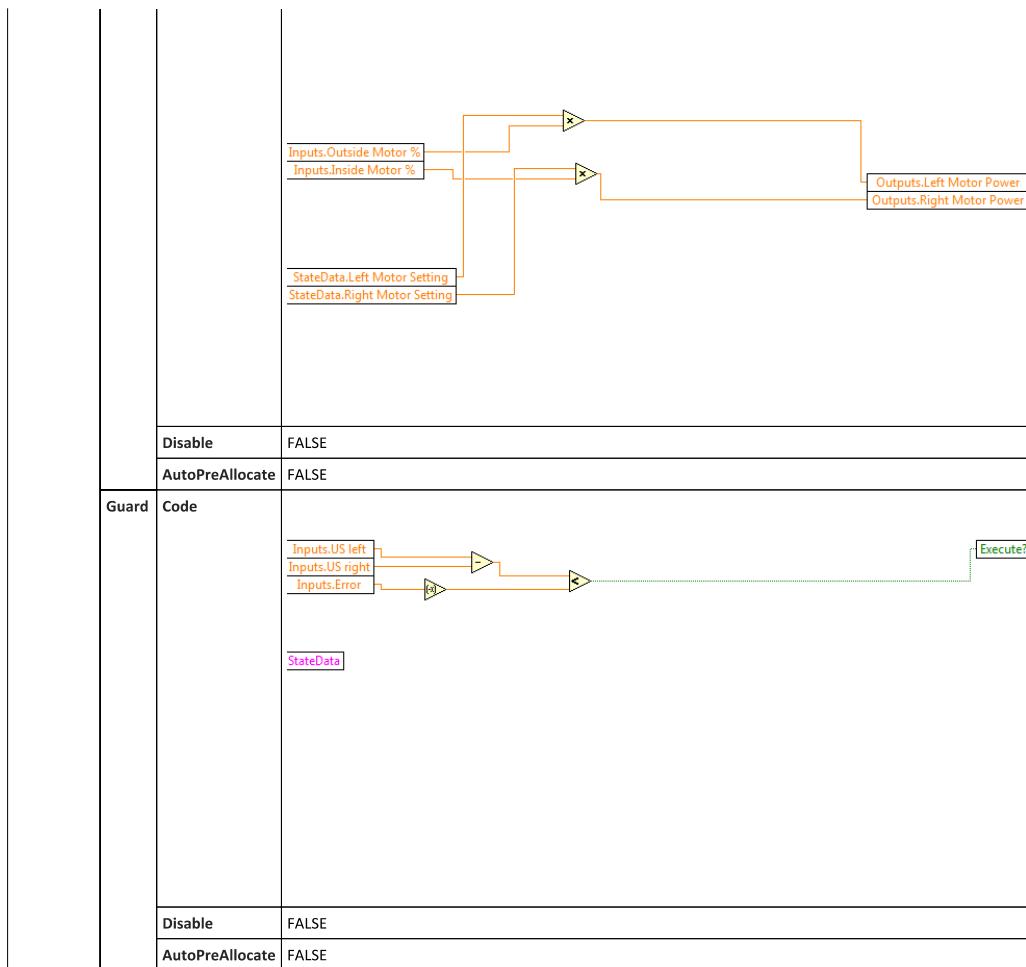
Description		-
Source		<u>Turn Left</u>
Destination		<u>Straight</u>
Containing Region		<u>Region</u>
Transition Code	Description	
	Trigger List	NULL
Action	Code	

**C.1.B.1.M Transition**

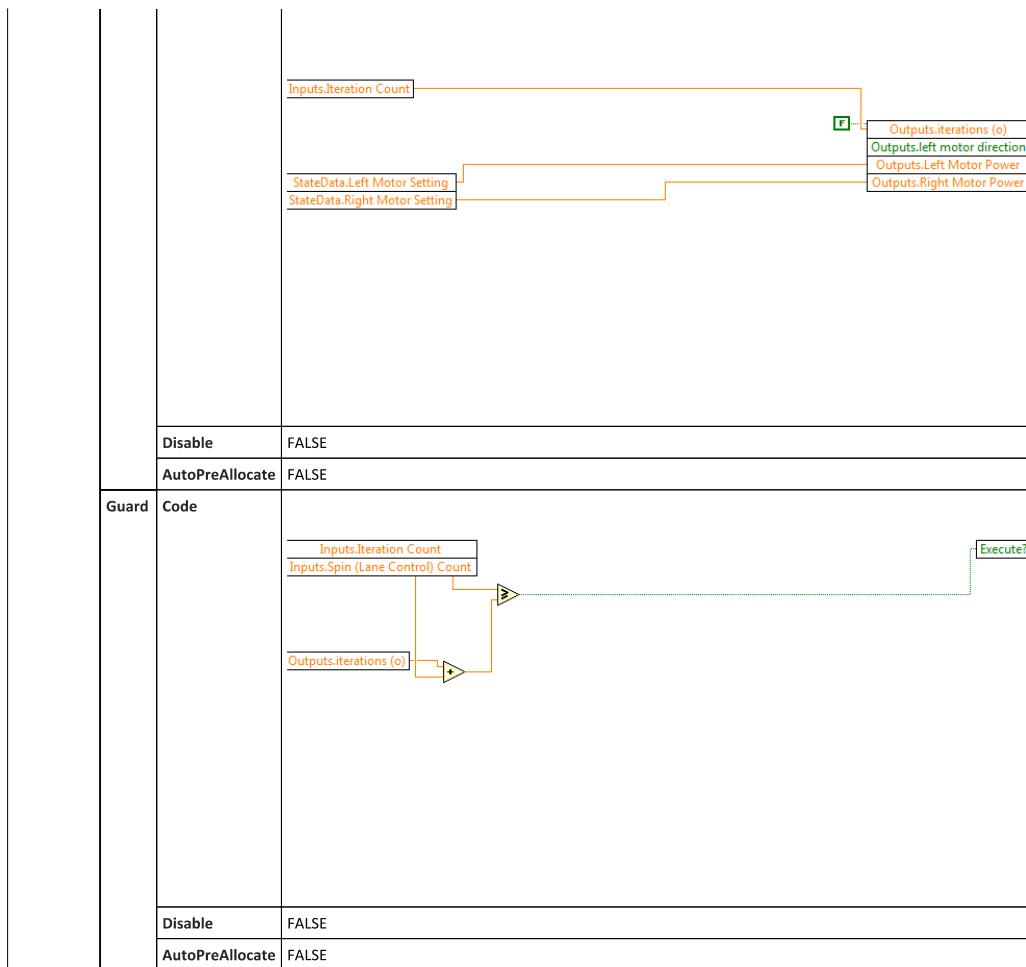
Description		-
Source		Turn Right
Destination		Straight
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.B.1.N Transition**

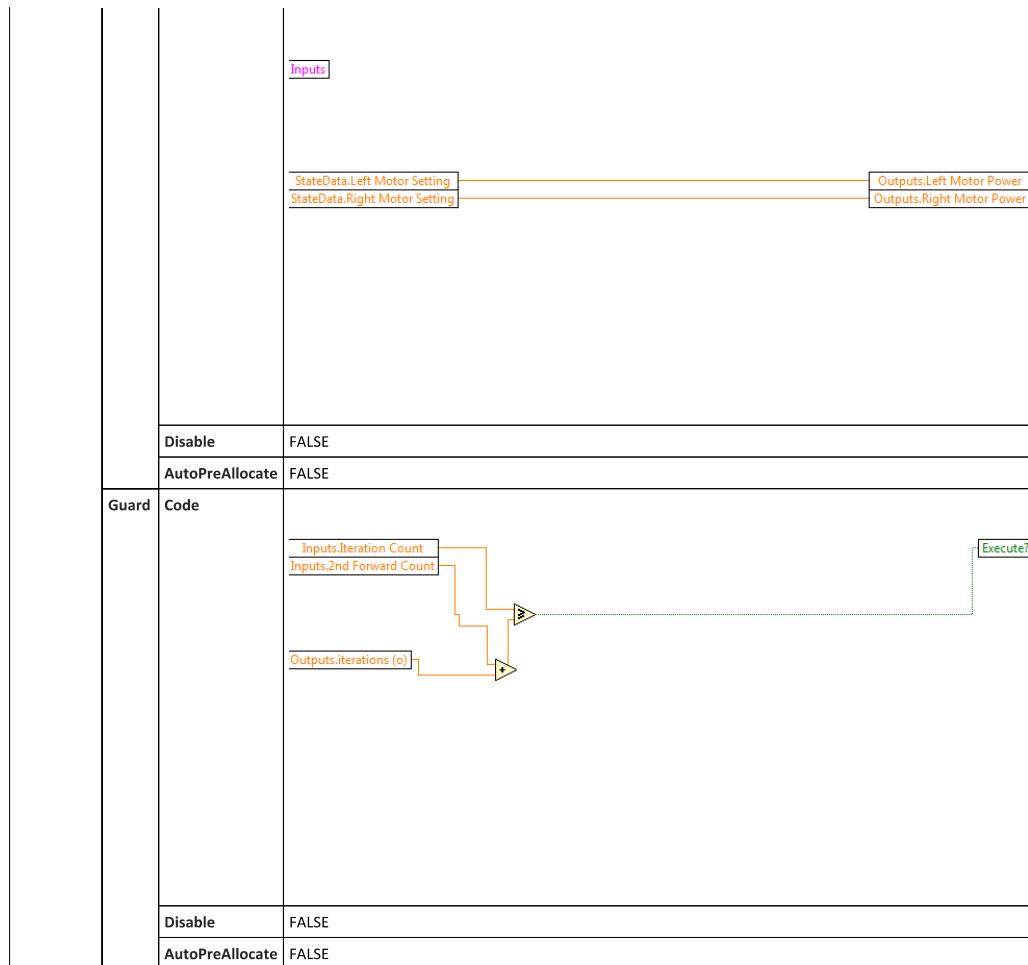
Description	-						
Source	Straight						
Destination	Turn Right						
Containing Region	Region						
Transition Code	<table border="1"> <tr> <td>Description</td> <td></td> </tr> <tr> <td>Trigger List</td> <td>NULL</td> </tr> <tr> <td>Action</td> <td>Code</td> </tr> </table>	Description		Trigger List	NULL	Action	Code
Description							
Trigger List	NULL						
Action	Code						

**C.1.B.1.O Transition**

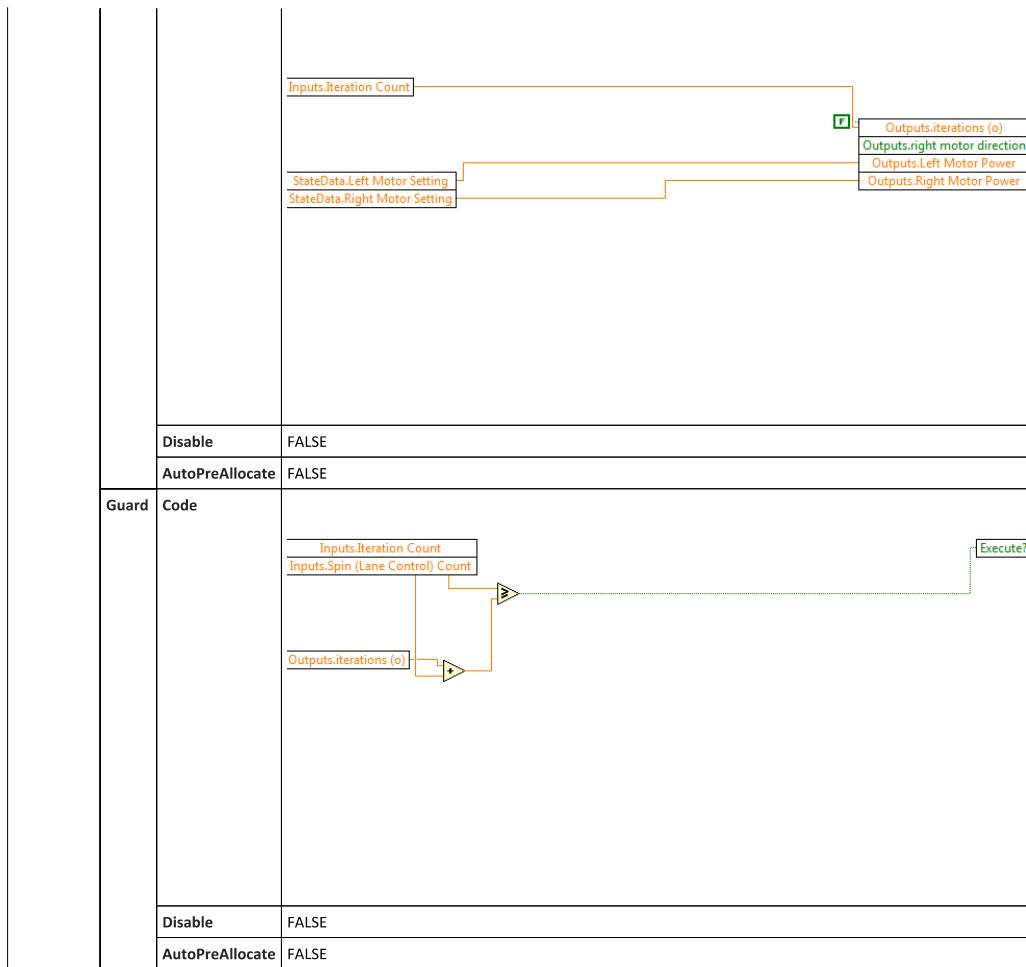
Description		-
Source		Spin Left
Destination		Forward_2
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.B.1.P Transition**

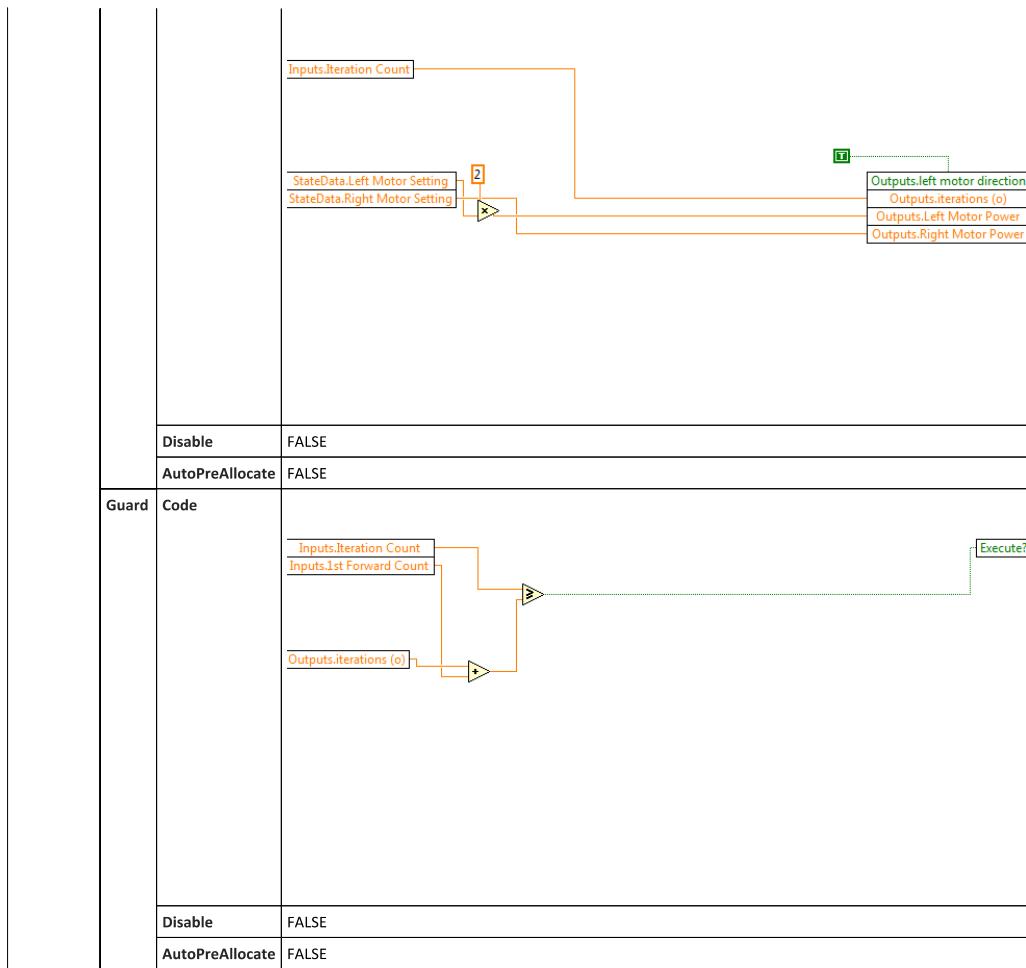
Description	-						
Source	Forward 2						
Destination	Straight						
Containing Region	Region						
Transition Code	<table border="1"> <tr> <td>Description</td> <td></td> </tr> <tr> <td>Trigger List</td> <td>NULL</td> </tr> <tr> <td>Action</td> <td>Code</td> </tr> </table>	Description		Trigger List	NULL	Action	Code
Description							
Trigger List	NULL						
Action	Code						

**C.1.B.1.Q Transition**

Description		-
Source		Spin Right
Destination		Forward 2
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.B.1.R Transition**

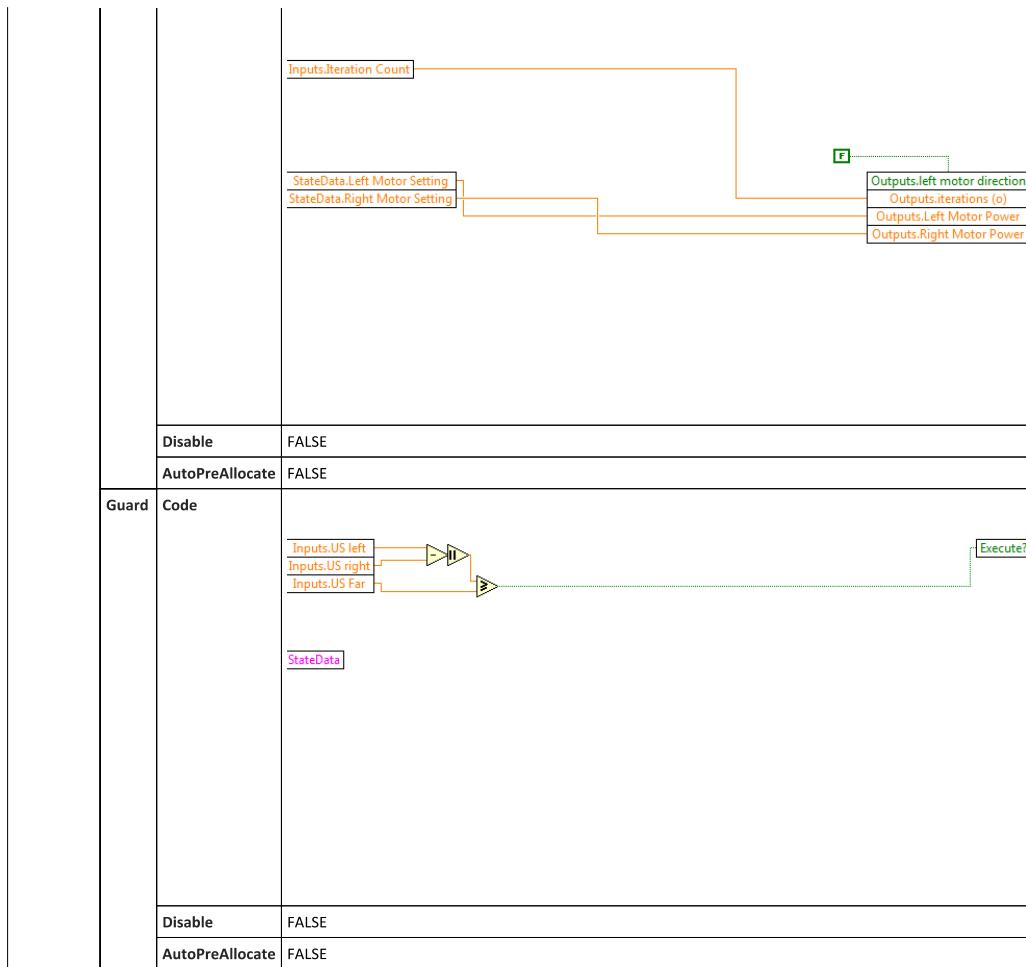
Description		-
Source		Forward to LS
Destination		Spin Left
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action Code	



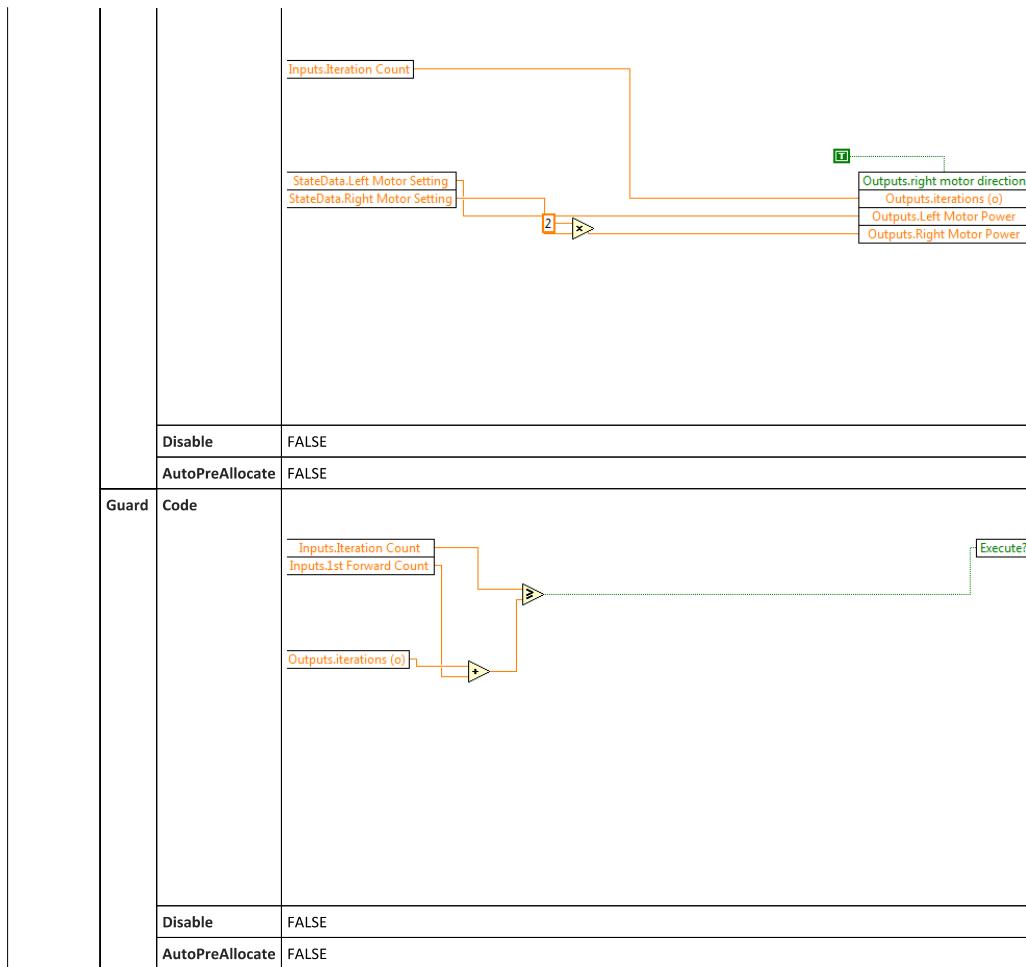
C.1.B.1.S Transition



Description		-
Source		Turn Left
Destination		Forward to LS
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action Code	

**C.1.B.1.T Transition**

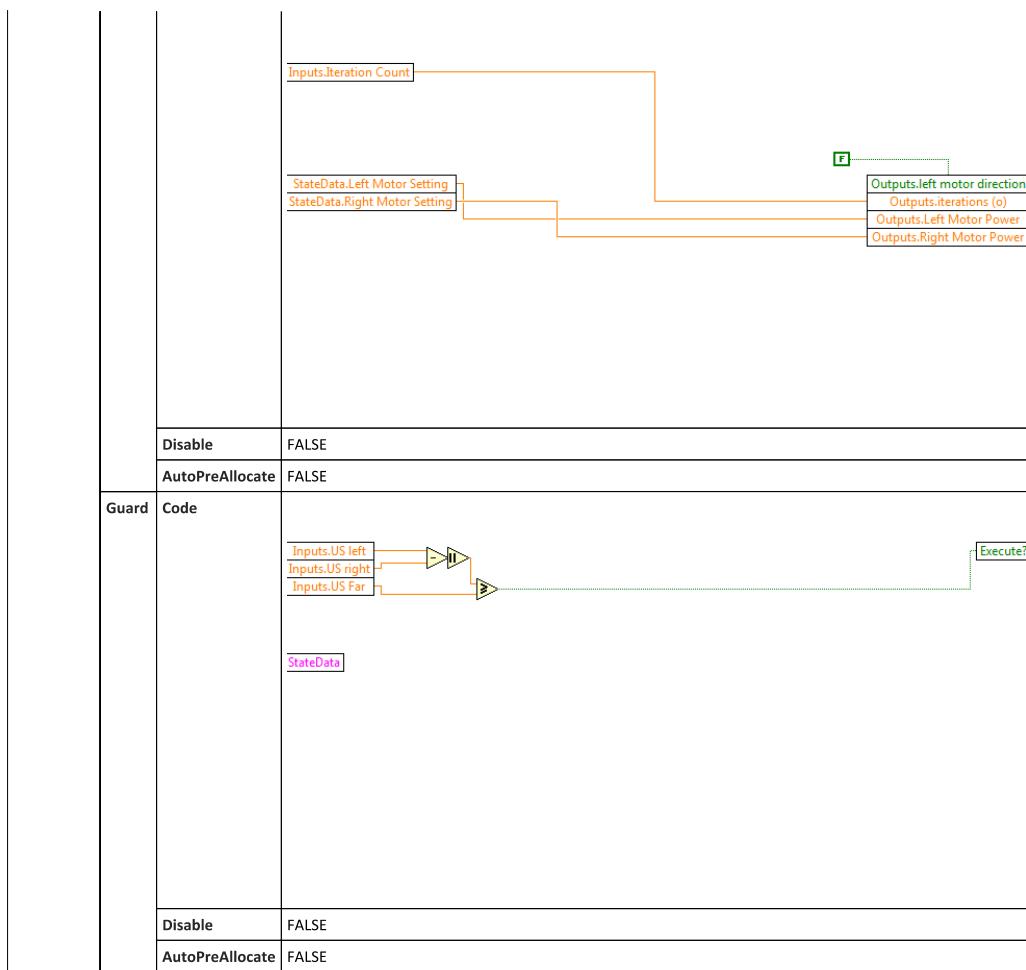
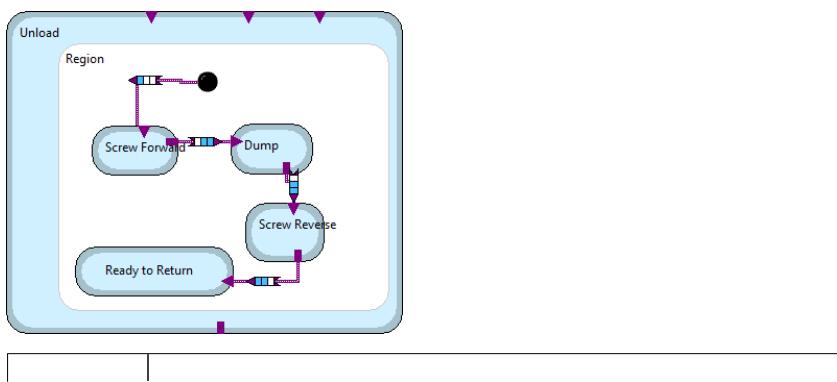
Description		-
Source		Forward to RS
Destination		Spin Right
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code



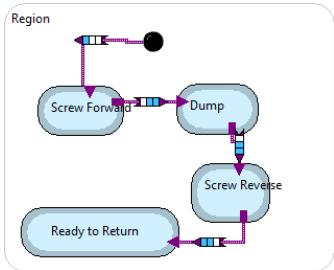
C.1.B.1.U Transition



Description		-
Source		Turn Right
Destination		Forward to RS
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action Code	

**C.1.C Unload**

Description	-
Source	Uphill Movement -> Transition , Only Left Bumper Hit -> Transition , Only Right Bumper Hit -> Transition
Destination	Transition -> Turn Around
Containing Region	Region
Contained Regions	Region

C.1.C.1 Region

Description	-
Containing State	Unload
Contained Objects	Initial , Screw Forward , Dump , Screw Reverse , Ready to Return
Contained Transitions	Transition , Transition , Transition , Transition

C.1.C.1.A Initial

Description	-
Destination	Transition -> Screw Forward
Containing Region	Region

C.1.C.1.B Screw Forward

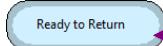
Description	-
Source	Initial -> Transition
Destination	Transition -> Dump
Containing Region	Region

C.1.C.1.C Dump

Description	-
Source	Screw Forward -> Transition
Destination	Transition -> Screw Reverse
Containing Region	Region

C.1.C.1.D Screw Reverse

Description	-
Source	Dump ->Transition
Destination	Transition ->Ready to Return
Containing Region	Region

C.1.C.1.E Ready to Return

Description	-
Source	Screw Reverse ->Transition
Destination	
Containing Region	Region

C.1.C.1.F Transition

Description	-														
Source	Initial														
Destination	Screw Forward														
Containing Region	Region														
Transition Code	<table border="1"> <tr> <td colspan="2">Description</td> </tr> <tr> <td colspan="2">Trigger List</td> </tr> <tr> <td colspan="2">{Responds to all triggers}</td> </tr> <tr> <td>Action</td> <td>Code</td> </tr> <tr> <td colspan="2"> <div style="border: 1px solid black; padding: 5px;">Inputs:Lead Screw setpoint</div> <div style="border: 1px solid black; padding: 5px; margin-top: 10px;">Outputs</div> </td> </tr> <tr> <td>Disable</td> <td>FALSE</td> </tr> <tr> <td>AutoPreAllocate</td> <td>FALSE</td> </tr> </table>	Description		Trigger List		{Responds to all triggers}		Action	Code	<div style="border: 1px solid black; padding: 5px;">Inputs:Lead Screw setpoint</div> <div style="border: 1px solid black; padding: 5px; margin-top: 10px;">Outputs</div>		Disable	FALSE	AutoPreAllocate	FALSE
Description															
Trigger List															
{Responds to all triggers}															
Action	Code														
<div style="border: 1px solid black; padding: 5px;">Inputs:Lead Screw setpoint</div> <div style="border: 1px solid black; padding: 5px; margin-top: 10px;">Outputs</div>															
Disable	FALSE														
AutoPreAllocate	FALSE														

C.1.C.1.G Transition

Description	-
-------------	---

Source	Screw Forward			
Destination	<u>Dump</u>			
Containing Region	<u>Region</u>			
Transition Code	<p>Description</p> <p>Trigger List</p> <p>NULL</p>			
Action	<table border="1"> <tr> <td>Code</td> <td> <p>Inputs.Iteration Count</p> <p>Outputs</p> <p>Outputs.iterations (o)</p> <p>Outputs.Lead Screw on</p> </td> </tr> </table>		Code	<p>Inputs.Iteration Count</p> <p>Outputs</p> <p>Outputs.iterations (o)</p> <p>Outputs.Lead Screw on</p>
Code	<p>Inputs.Iteration Count</p> <p>Outputs</p> <p>Outputs.iterations (o)</p> <p>Outputs.Lead Screw on</p>			
Disable	FALSE			
AutoPreAllocate	FALSE			
Guard	<table border="1"> <tr> <td>Code</td> <td> <p>Inputs.Lead screw contact</p> <p>Execute?</p> <p>StateData</p> </td> </tr> </table>		Code	<p>Inputs.Lead screw contact</p> <p>Execute?</p> <p>StateData</p>
Code	<p>Inputs.Lead screw contact</p> <p>Execute?</p> <p>StateData</p>			
Disable	FALSE			
AutoPreAllocate	FALSE			

C.1.C.1.H Transition

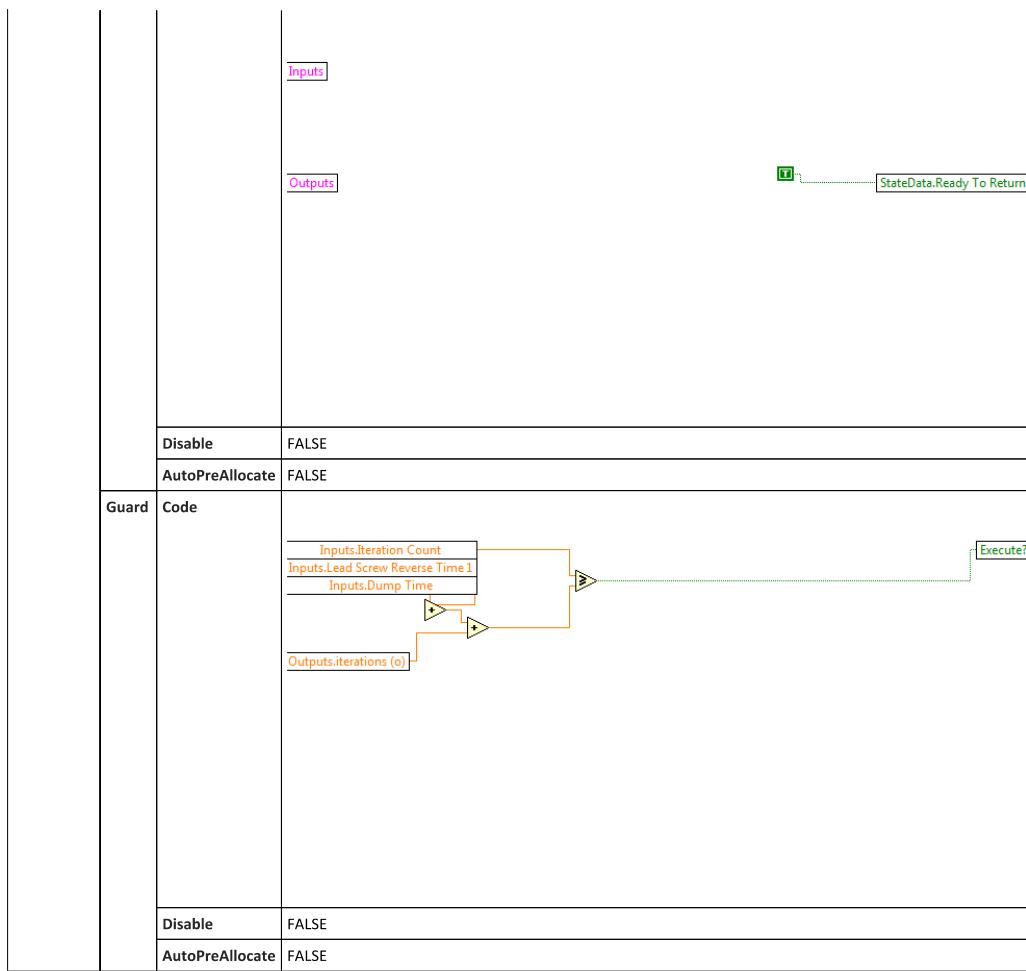


Description	-
Source	<u>Dump</u>
Destination	<u>Screw Reverse</u>
Containing Region	<u>Region</u>

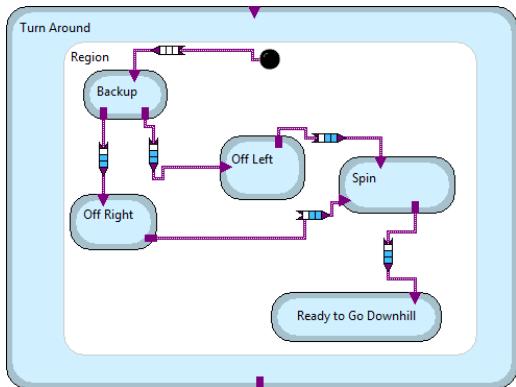
Transition Code	Description	
	Trigger List	NULL
	Action Code	<pre> graph LR A[Inputs.Lead Screw setpoint] --> S1(()) S1 --> B[Outputs] B --> P1[Parallel Block] P1 --> C[Outputs.Lead Screw Direction] P1 --> C[Outputs.Lead Screw on] P1 --> D[Outputs.Lead screw power] P1 --> E[Outputs.Left Motor Power] P1 --> F[Outputs.Right Motor Power] G[0] --- D </pre>
	Disable	FALSE
	AutoPreAllocate	FALSE
	Guard Code	<pre> graph LR A[Inputs.Iteration Count] --- S2(()) B[Inputs.Dump Time] --- S2 S2 --> P2[Parallel Block] P2 --> C2[Switch] P2 --> D2[Outputs.iterations (o)] G2[0] --- C2 E2[Execute?] --- S2 </pre>
	Disable	FALSE
	AutoPreAllocate	FALSE

C.1.C.1.I Transition

	Description	-
	Source	Screw Reverse
	Destination	Ready to Return
	Containing Region	Region
Transition Code	Description	
	Trigger List	NULL
	Action Code	

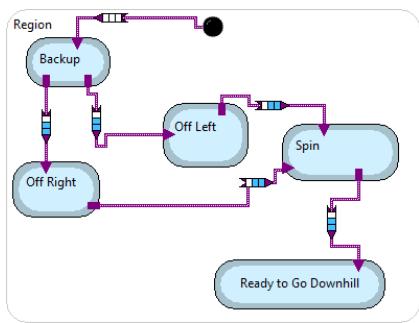


C.1.D Turn Around



Description	-
Source	Unload ->Transition
Destination	Transition ->Downhill
Containing Region	Region
Contained Regions	Region

C.1.D.1 Region



Description	-
Containing State	Turn Around
Contained Objects	Initial, Backup, Spin, Off Right, Off Left, Ready to Go Downhill
Contained Transitions	Transition, Transition, Transition, Transition, Transition, Transition

C.1.D.1.A Initial



Description	-
Destination	Transition ->Backup
Containing Region	Region

C.1.D.1.B Backup



Description	-
Source	Initial -> Transition
Destination	Transition -> Off Right , Transition -> Off Left
Containing Region	Region

C.1.D.1.C Spin

Description	-
Source	Off Right -> Transition , Off Left -> Transition
Destination	Transition -> Ready to Go Downhill
Containing Region	Region

C.1.D.1.D Off Right

Description	-
Source	Backup -> Transition
Destination	Transition -> Spin
Containing Region	Region

C.1.D.1.E Off Left

Description	-
Source	Backup -> Transition
Destination	Transition -> Spin
Containing Region	Region

C.1.D.1.F Ready to Go Downhill

Description	-
Source	Spin -> Transition
Destination	
Containing Region	Region

C.1.D.1.G Transition

Description	-
-------------	---

Source	Initial
Destination	Backup
Containing Region	Region
Transition Code	Description
	Trigger List {Responds to all triggers}

C.1.D.1.H Transition

Description		-
Source		Off Right
Destination		Spin
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
Action	Code	<p>Inputs.Iteration Count</p> <p>Turns around by reversing the direction of the right motor.</p> <p>StateData.Left Motor Setting</p> <p>StateData.Right Motor Setting</p> <p>Outputs.iterations (o)</p> <p>Outputs.left motor direction</p> <p>Outputs.Left Motor Power</p> <p>Outputs.Right Motor Power</p>
Disable	Code	FALSE
AutoPreAllocate	Code	FALSE
Guard	Code	<p>Inputs.Iteration Count</p> <p>Inputs.Backup (Turnaround) Count</p> <p>Execute?</p> <p>Outputs.iterations (o)</p> <p>Backup for a count of 25</p>
Disable	Code	FALSE

	AutoPreAllocate	FALSE
--	-----------------	-------

C.1.D.1.I Transition

Description		-
Source		Backup
Destination		Off Right
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
Action	Code	<pre> graph LR IC[Inputs Iteration Count] --> AND1(()) BTOEE[Inputs Backup Turnaround Outer Motor Effort] --> AND1 BTIE[Inputs Backup Turnaround Inner Motor Effort] --> AND1 LMS[StateData Left Motor Setting] --> AND2(()) RMS[StateData Right Motor Setting] --> AND2 AND1 --> AND2 AND2 --> Reverse[Reverse Both Motors] Reverse --> OM[Outputs motors on] Reverse --> OLM[Outputs left motor direction] Reverse --> ORM[Outputs right motor direction] Reverse --> OI[Outputs iterations o] Reverse --> LMP[Outputs Left Motor Power] Reverse --> RMP[Outputs Right Motor Power] </pre>
	Disable	FALSE
	AutoPreAllocate	FALSE
Guard	Code	<pre> graph TD USL[Inputs US left] --> Decision{ } USR[Inputs US right] --> Decision SD[StateData] --> Decision Decision --> Execute[Execute?] </pre>
	Disable	FALSE
	AutoPreAllocate	FALSE

C.1.D.1.J Transition



Description		-
Source		Backup
Destination		Off Left
Containing Region		Region
Transition Code		
Trigger List		NULL
Action	Code	<pre> graph LR A[Inputs.Iteration Count] --> B[Inputs.Backup (Turnaround) Outer Motor Effort] A --> C[Inputs.Backup (Turnaround) Inner Motor Effort] B --> D[StateData.Left Motor Setting] C --> E[StateData.Right Motor Setting] D --> F(()) E --> G(()) F --> H(()) G --> I(()) H --> J(()) I --> K(()) J --> L(()) K --> M[Reverse Both Motors] L --> N[Outputs.motors on] M --> O[Outputs.left motor direction] M --> P[Outputs.right motor direction] M --> Q[Outputs.iterations (o)] M --> R[Outputs.Left Motor Power] M --> S[Outputs.Right Motor Power] </pre>
		Disable FALSE AutoPreAllocate FALSE
Guard	Code	<pre> graph LR A[Inputs.US left] --> B[Inputs.US right] A --> C[StateData] B --> D(()) C --> E(()) D --> F(()) E --> G(()) F --> H(()) G --> I(()) H --> J(()) I --> K[Execute?] </pre>
		Disable FALSE AutoPreAllocate FALSE

C.1.D.1.K Transition



Description		-
Source		Off Left
Destination		Spin

Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action Code	<pre> graph LR A[Inputs.Iteration Count] --> B["Turns around by reversing the direction of the left motor."] B --> C[StateData.Left Motor Setting] B --> D[StateData.Right Motor Setting] C --> E[Outputs.right motor direction] D --> F[Outputs.right motor direction] E --> G[Outputs.left motor power] F --> G G --> H[Outputs.right motor power] </pre>
	Disable	FALSE
	AutoPreAllocate	FALSE
	Guard Code	<pre> graph LR A[Inputs.Iteration Count] --> B[Inputs.Backup (Turnaround) Count] B --> C{Execute?} C --> D[Outputs.iterations (o)] C --> E </pre>
	Disable	FALSE
	AutoPreAllocate	FALSE

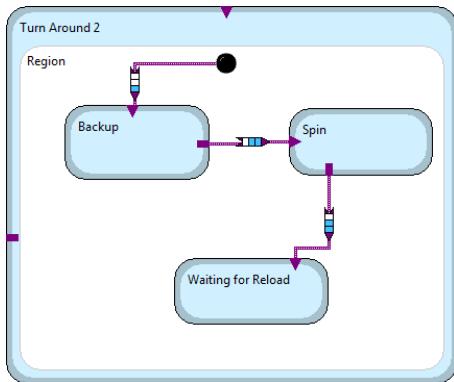
C.1.D.1.L Transition



Description		-
Source		Spin
Destination		Ready to Go Downhill
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL

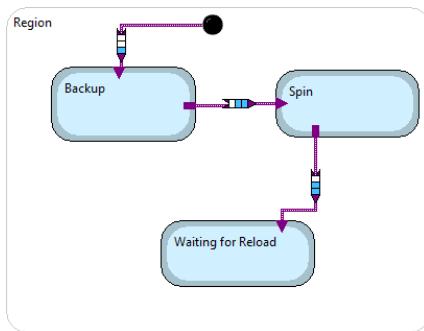
Action	Code
	<p>Inputs.Downhill Reduction</p> <p>Outputs</p> <pre> graph LR A[Inputs.Downhill Reduction] --> B[Outputs] B --> C[StateData.Downhill Reached] </pre>
Disable	FALSE
AutoPreAllocate	FALSE
Guard	Code
	<pre> graph LR A[Inputs.Iteration Count] --> B[Inputs.Spin Turnaround Count] B --> C[Inputs.Backup Turnaround Count] C --> D[Iterations o] C --> E[Execute?] D --> E </pre>
Disable	FALSE
AutoPreAllocate	FALSE

C.1.E Turn Around 2



Description	-
Source	Downhill ->Transition
Destination	Transition ->Uphill Movement
Containing Region	Region
Contained Regions	Region

C.1.E.1 Region



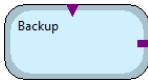
Description	-
Containing State	Turn Around 2
Contained Objects	Initial, Backup, Spin, Waiting for Reload
Contained Transitions	Transition, Transition, Transition

C.1.E.1.A Initial



Description	-
Destination	Transition ->Backup
Containing Region	Region

C.1.E.1.B Backup



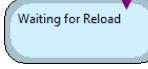
Description	-
Source	Initial ->Transition
Destination	Transition ->Spin
Containing Region	Region

C.1.E.1.C Spin



Description	-
Source	Backup ->Transition
Destination	Transition ->Waiting for Reload
Containing Region	Region

C.1.E.1.D Waiting for Reload

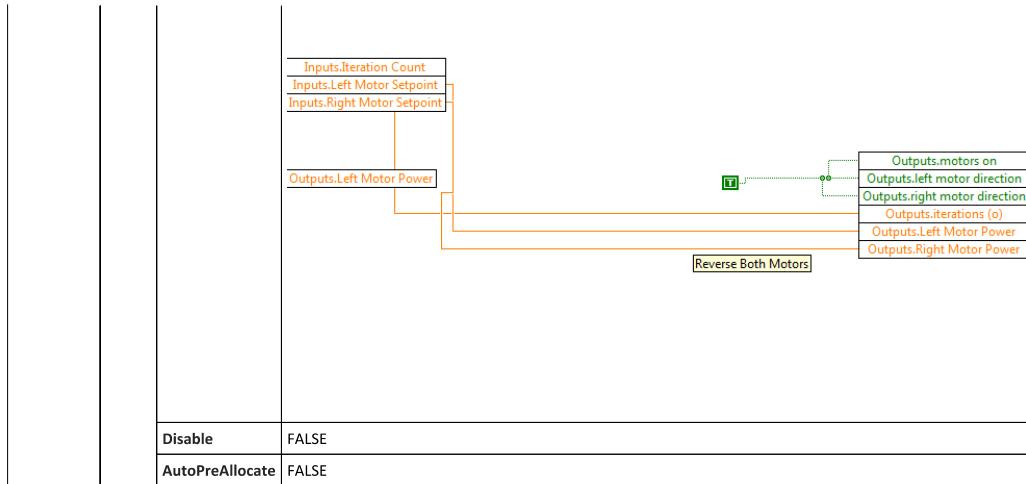


Description	-
Source	Spin ->Transition
Destination	
Containing Region	Region

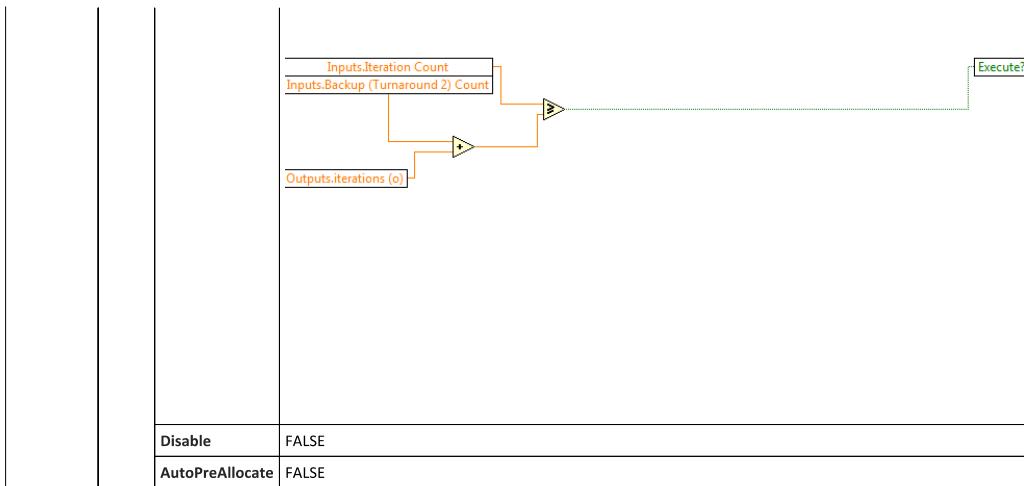
C.1.E.1.E Transition



Description		-
Source		Initial
Destination		Backup
Containing Region		Region
Transition Code	Description	
	Trigger List	{Responds to all triggers}
	Action	Code

**C.1.E.1.F Transition**

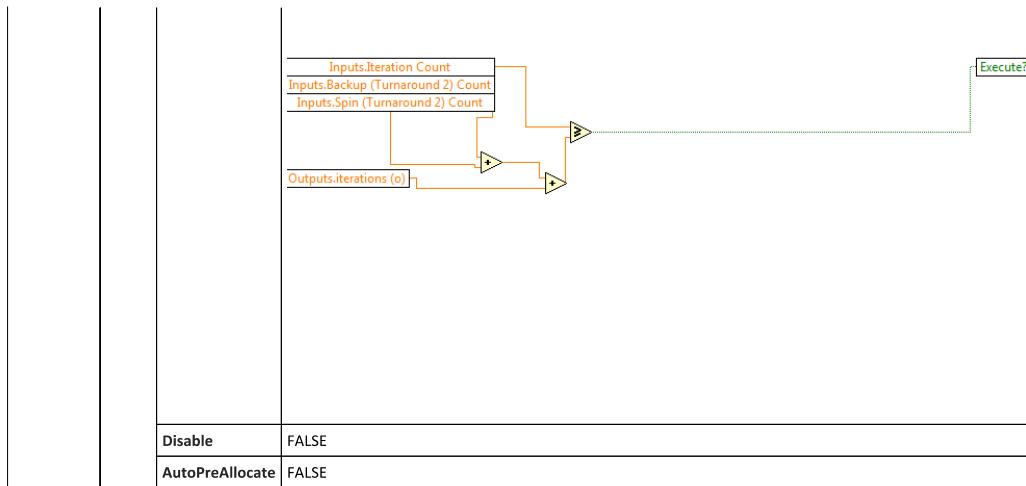
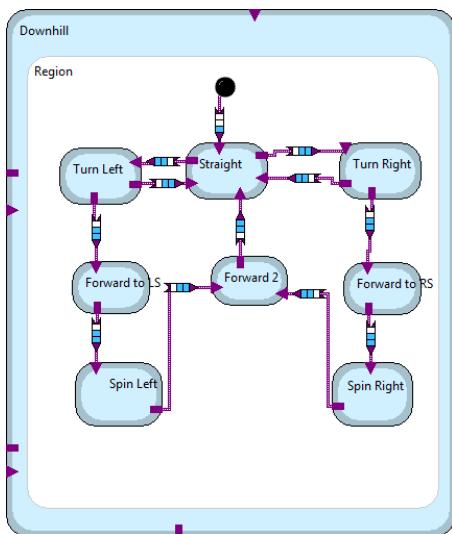
Description		-
Source		Backup
Destination		Spin
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
Action	Code	<p>Inputs.Left Motor Setpoint</p> <p>Inputs.Right Motor Setpoint</p> <p>StateData.Left Motor Setting</p> <p>Turns around by reversing the direction of the left motor.</p> <p>Outputs.right motor direction</p> <p>Outputs.Left Motor Power</p> <p>Outputs.Right Motor Power</p>
Guard	Disable	FALSE
	AutoPreAllocate	FALSE
Guard	Code	



C.1.E.1.G Transition

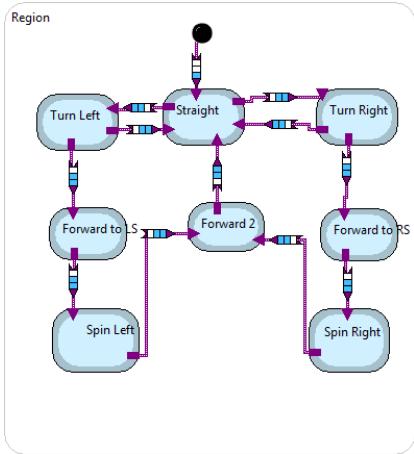


Description		-
Source		Spin
Destination		Waiting for Reload
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code
		 <pre> graph LR A[Inputs:Iteration Count] --> B[Outputs] </pre>
	Disable	FALSE
	AutoPreAllocate	FALSE
	Guard	Code

**C.1.F Downhill**

Description	-
Source	Turn Around -> Transition , Lead Screw Upright -> Transition , Speed Up Prior To End -> Transition
Destination	Transition -> Turn Around 2 , Transition -> Lead Screw Upright , Transition -> Speed Up Prior To End
Containing Region	Region
Contained Regions	Region

C.1.F.1 Region



Description	-
Containing State	Downhill
Contained Objects	Straight , Turn Right , Turn Left , Initial , Spin Left , Spin Right , Forward 2 , Forward to LS , Forward to RS
Contained Transitions	Transition , Transition

C.1.F.1.A Initial

Description	-
Destination	Transition >Straight
Containing Region	Region

C.1.F.1.B Straight

Description	-
Source	Initial ->Transition , Turn Left ->Transition , Turn Right ->Transition , Forward 2 ->Transition
Destination	Transition ->Turn Left , Transition ->Turn Right
Containing Region	Region

C.1.F.1.C Turn Right

Description	-
Source	Straight ->Transition
Destination	Transition ->Straight
Containing Region	Region

C.1.F.1.D Turn Left

Description	-
Source	Straight ->Transition
Destination	Transition ->Straight
Containing Region	Region

C.1.F.1.E Spin Left

Description	-
Source	Forward to LS ->Transition
Destination	Transition ->Forward 2
Containing Region	Region

C.1.F.1.F Spin Right

Description	-
Source	Forward to RS ->Transition
Destination	Transition ->Forward 2
Containing Region	Region

C.1.F.1.G Forward 2

Description	-
Source	Spin Left ->Transition, Spin Right ->Transition
Destination	Transition ->Straight
Containing Region	Region

C.1.F.1.H Forward to LS

Description	-
Source	Turn Left ->Transition
Destination	Transition ->Spin Left
Containing Region	Region

C.1.F.1.I Forward to RS



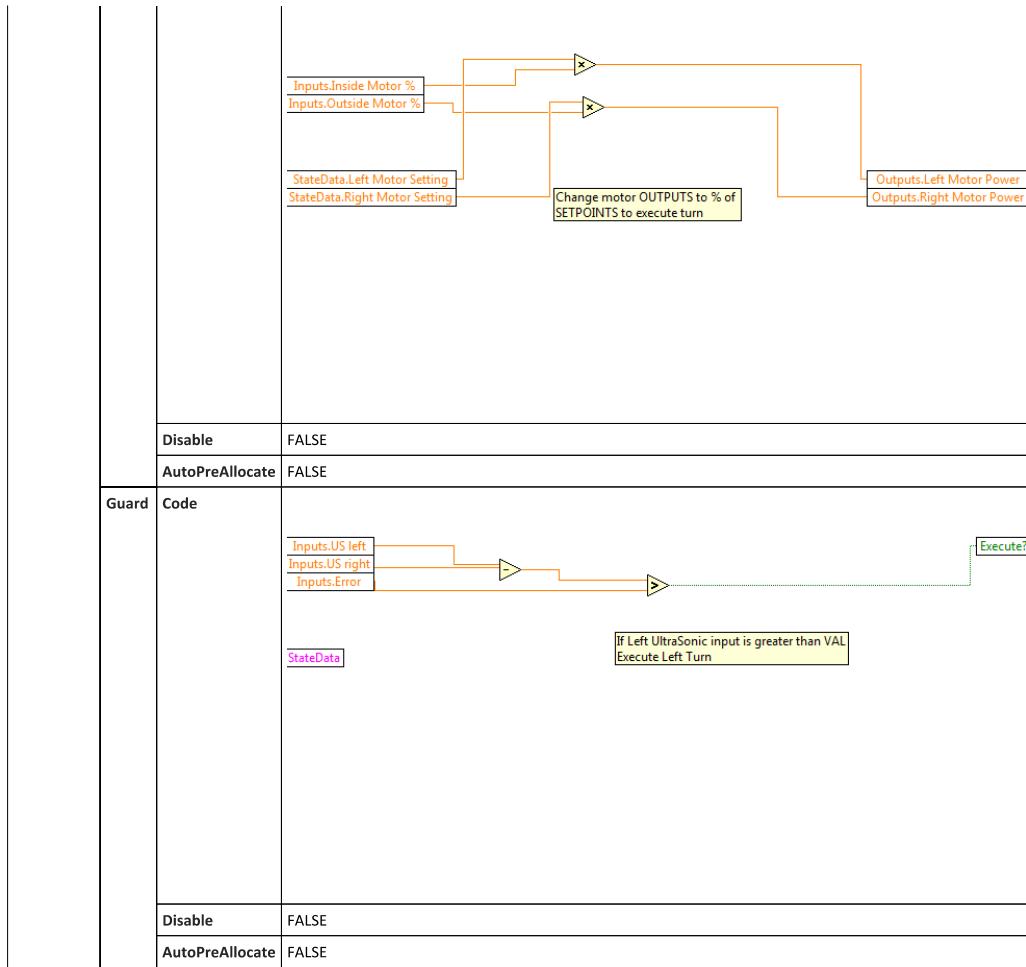
Description	-
Source	Turn Right ->Transition
Destination	Transition ->Spin Right
Containing Region	Region

C.1.F.1.J Transition

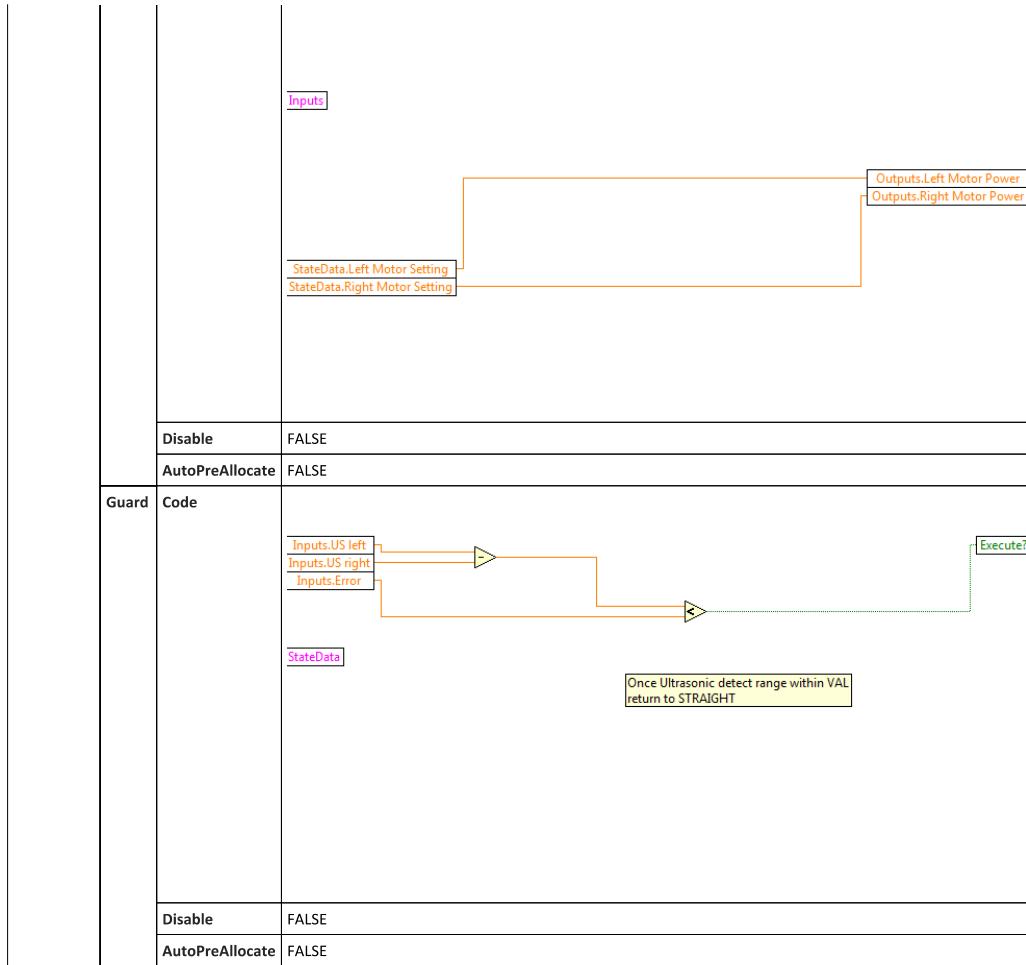
Description		-
Source		Initial
Destination		Straight
Containing Region		Region
Transition Code	Description	
	Trigger List	{Responds to all triggers}
	Action	Code
		<div style="border: 1px solid black; padding: 2px;">Inputs</div> <div style="display: flex; justify-content: space-around;"> <div style="border: 1px solid black; padding: 2px;">StateData.Left Motor Setting</div> <div style="border: 1px solid black; padding: 2px;">Outputs.Left Motor Power</div> </div> <div style="display: flex; justify-content: space-around;"> <div style="border: 1px solid black; padding: 2px;">StateData.Right Motor Setting</div> <div style="border: 1px solid black; padding: 2px;">Outputs.Right Motor Power</div> </div>
	Disable	FALSE
	AutoPreAllocate	FALSE

C.1.F.1.K Transition

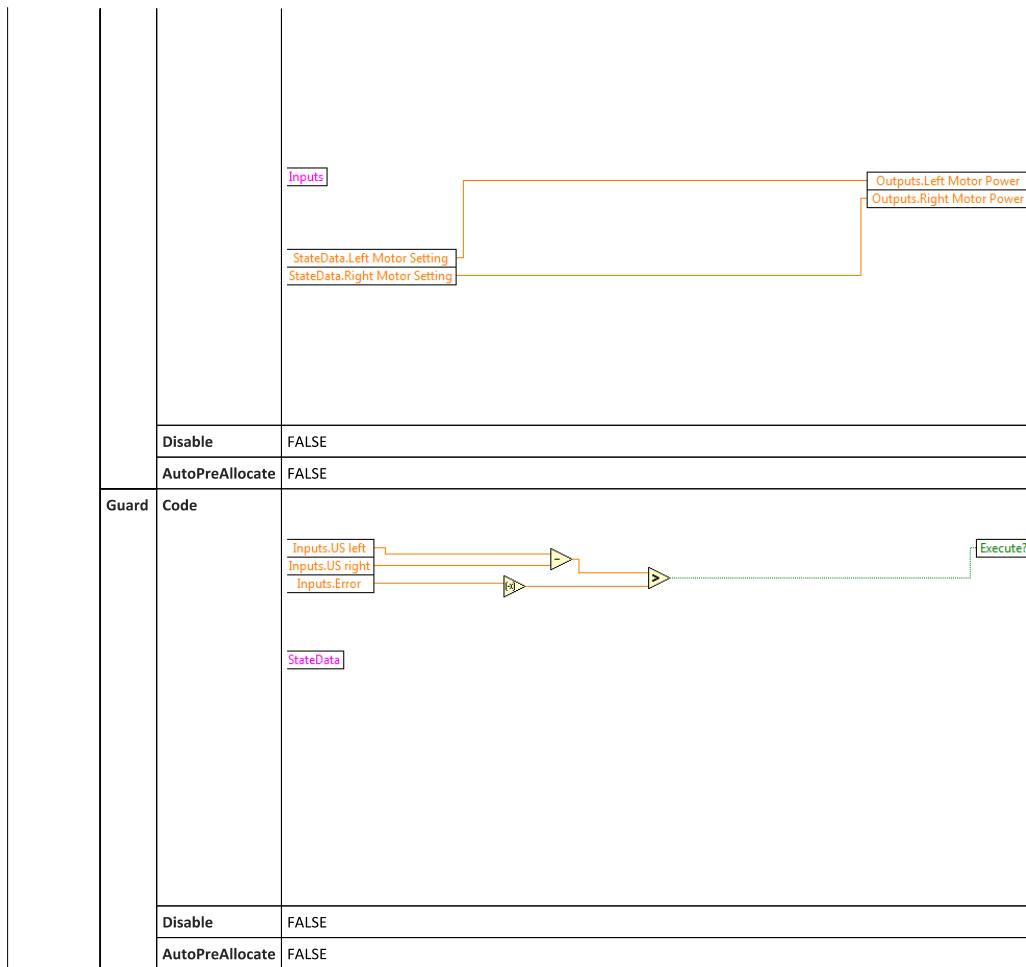
Description		-
Source		Straight
Destination		Turn Left
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.F.1.L Transition**

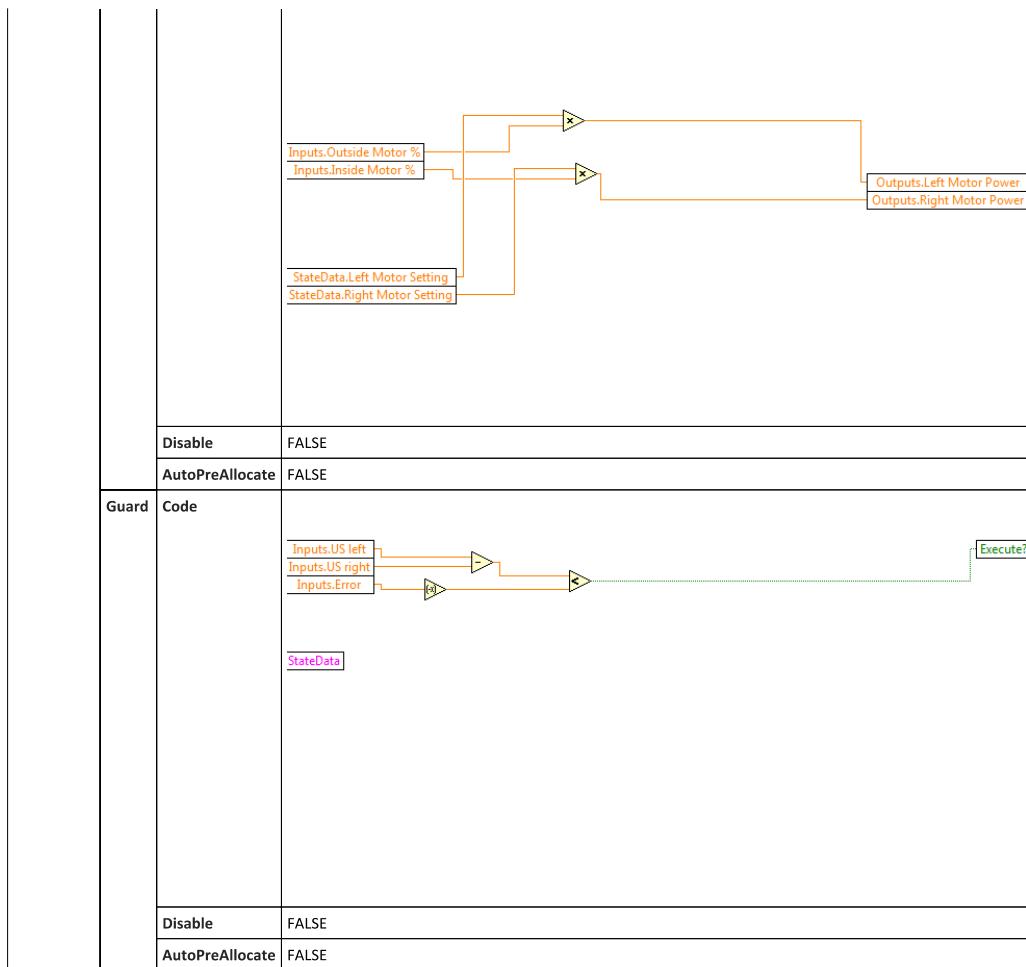
Description		-
Source		<u>Turn Left</u>
Destination		<u>Straight</u>
Containing Region		<u>Region</u>
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.F.1.M Transition**

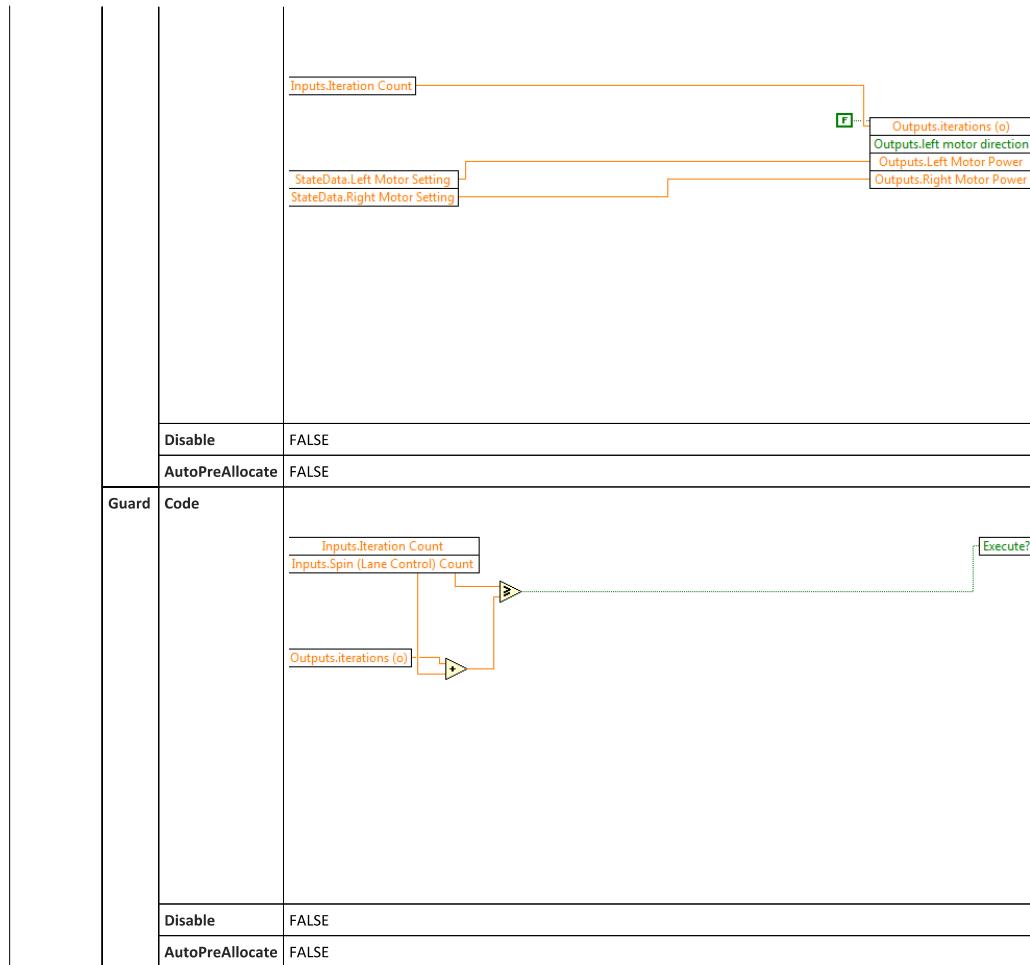
Description		-
Source		Turn Right
Destination		Straight
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
Action	Code	

**C.1.F.1.N Transition**

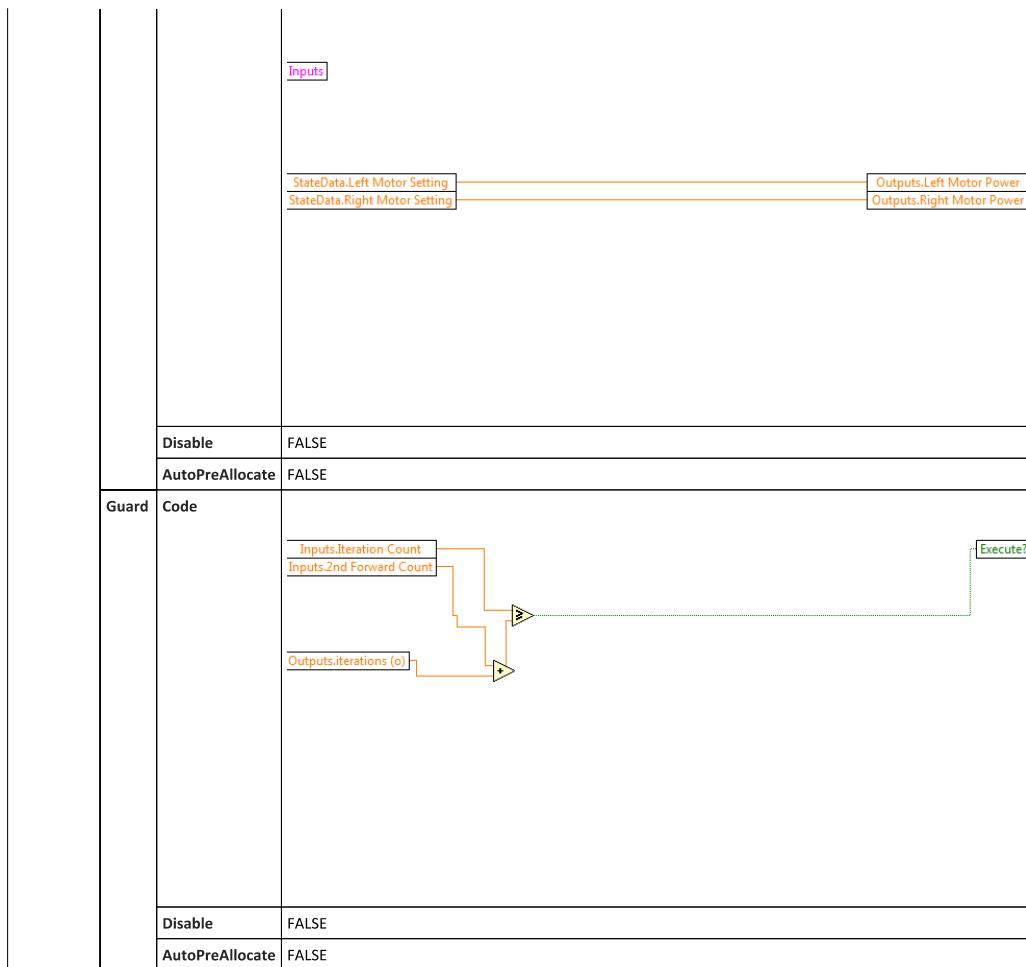
Description		-
Source		Straight
Destination		Turn Right
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.F.1.O Transition**

Description		-
Source		Spin Left
Destination		Forward 2
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.F.1.P Transition**

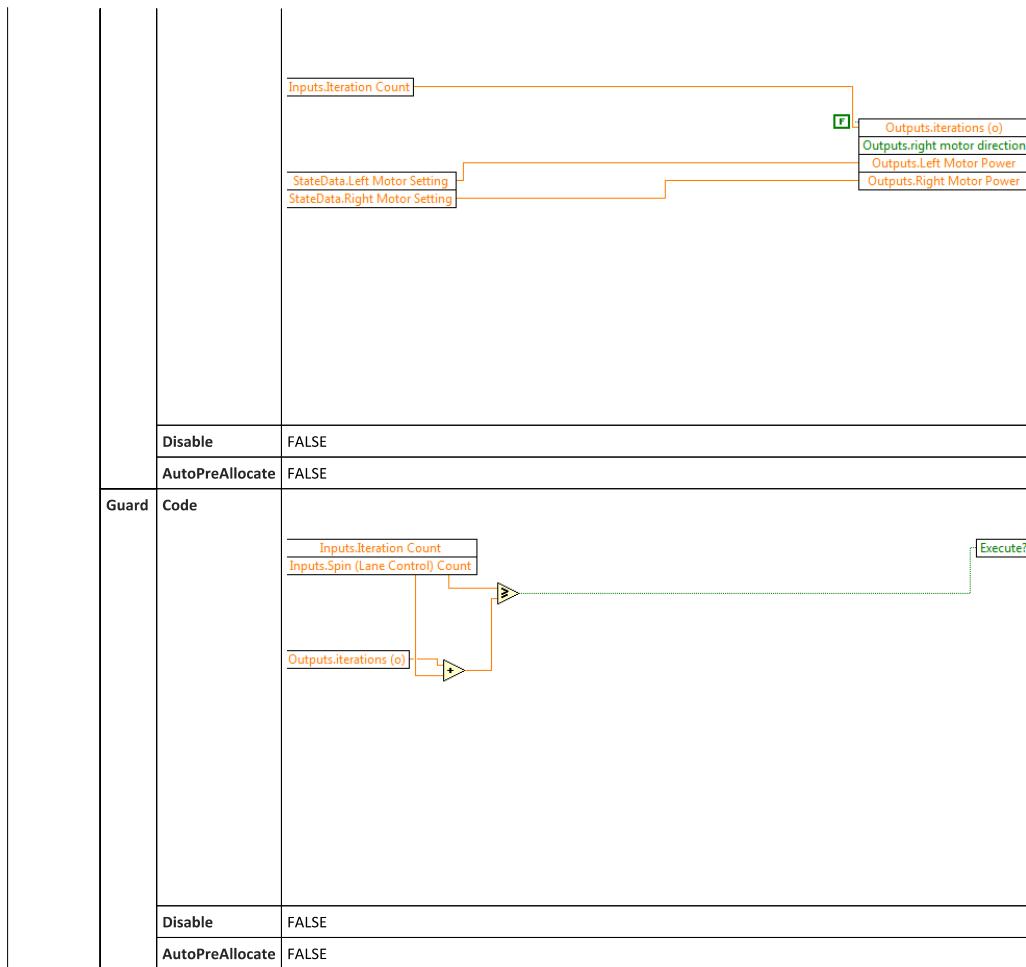
Description		-
Source		Forward 2
Destination		Straight
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
Action	Code	



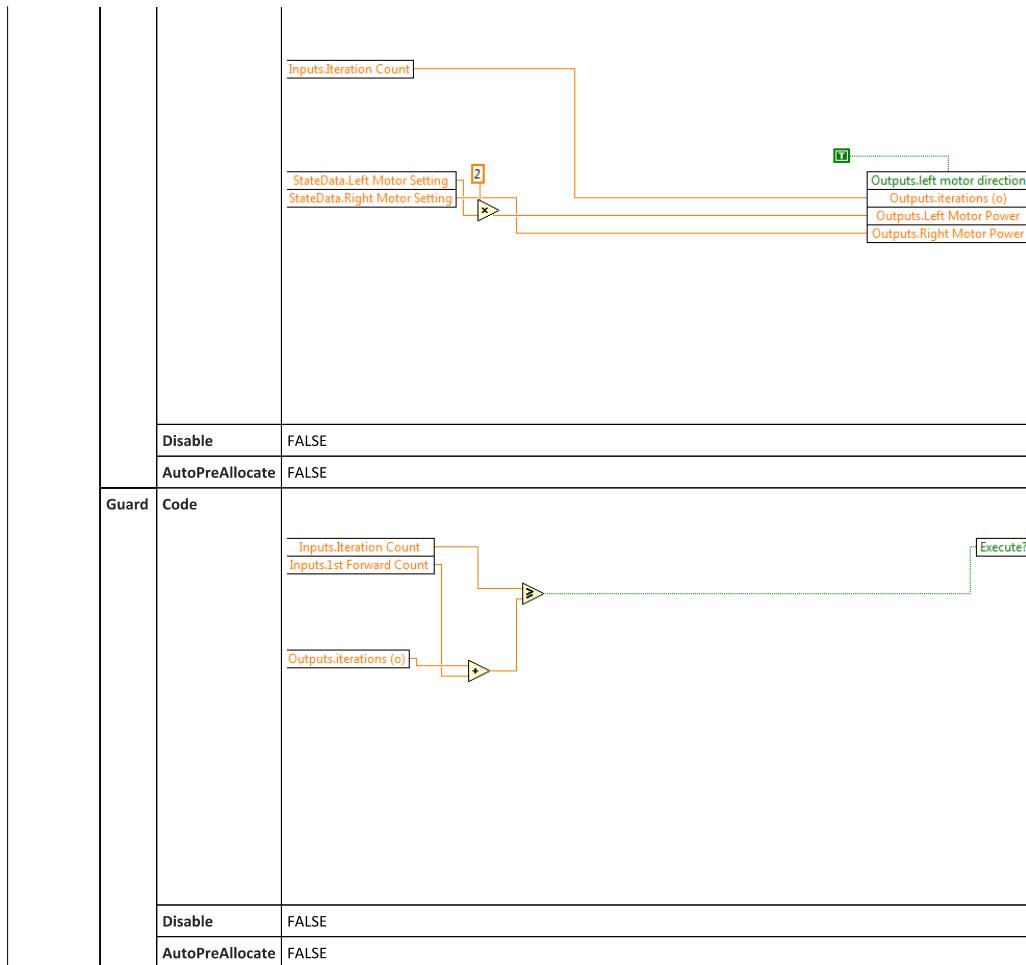
C.1.F.1.Q Transition



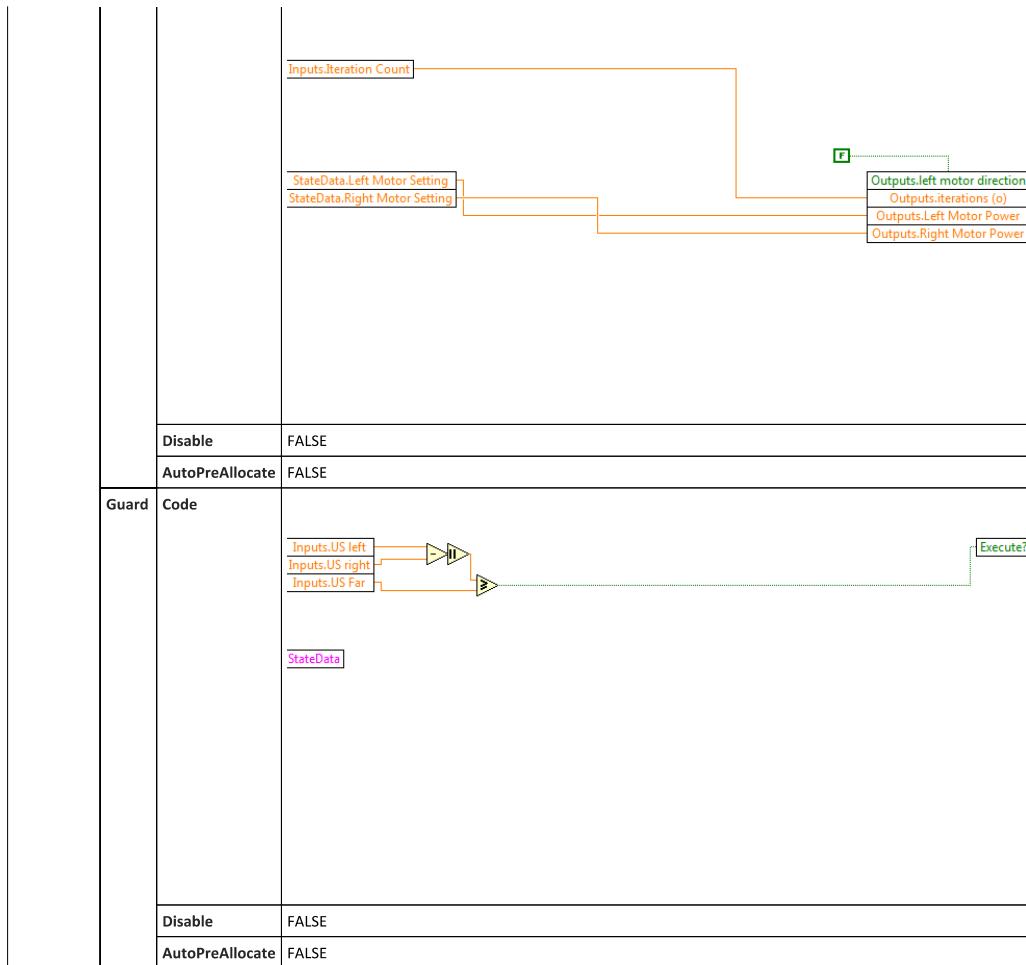
Description		-
Source		Spin Right
Destination		Forward_2
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.F.1.R Transition**

Description		-
Source		Forward to LS
Destination		Spin Left
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action Code	

**C.1.F.1.S Transition**

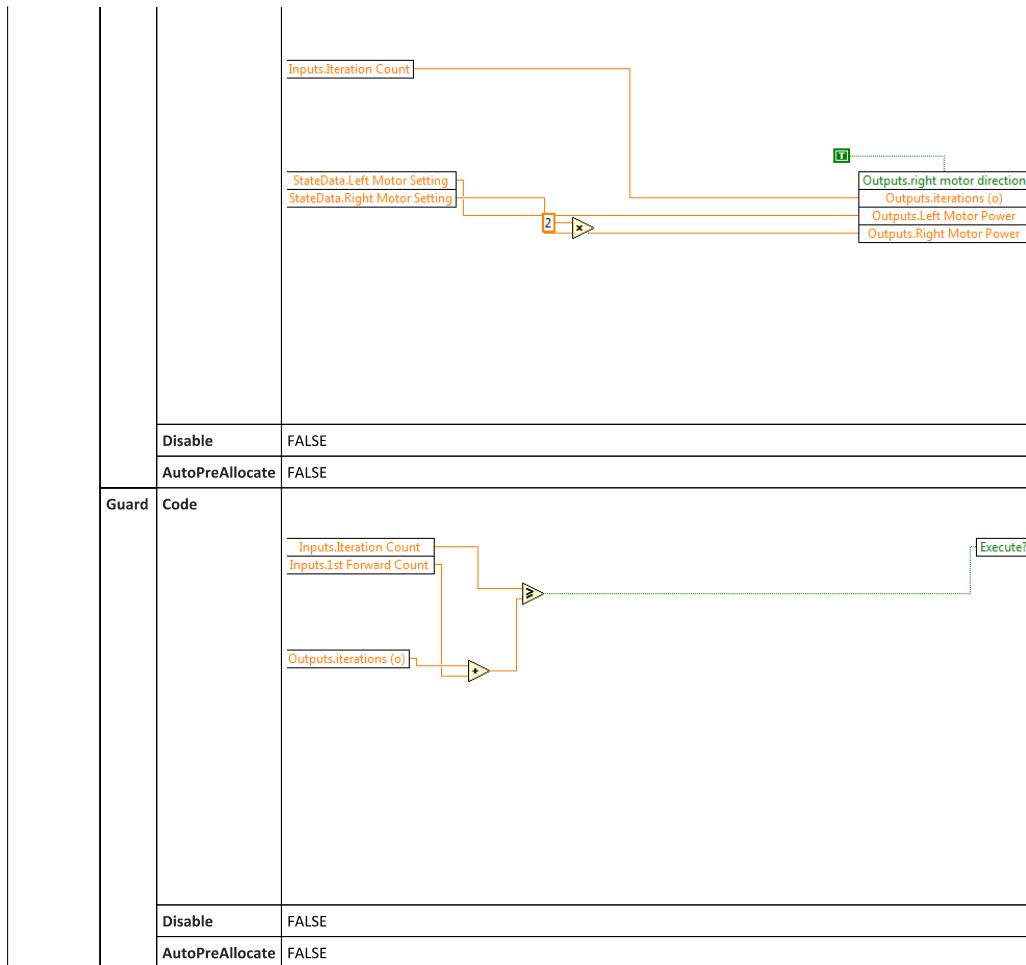
Description		-
Source		Turn Left
Destination		Forward to LS
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code



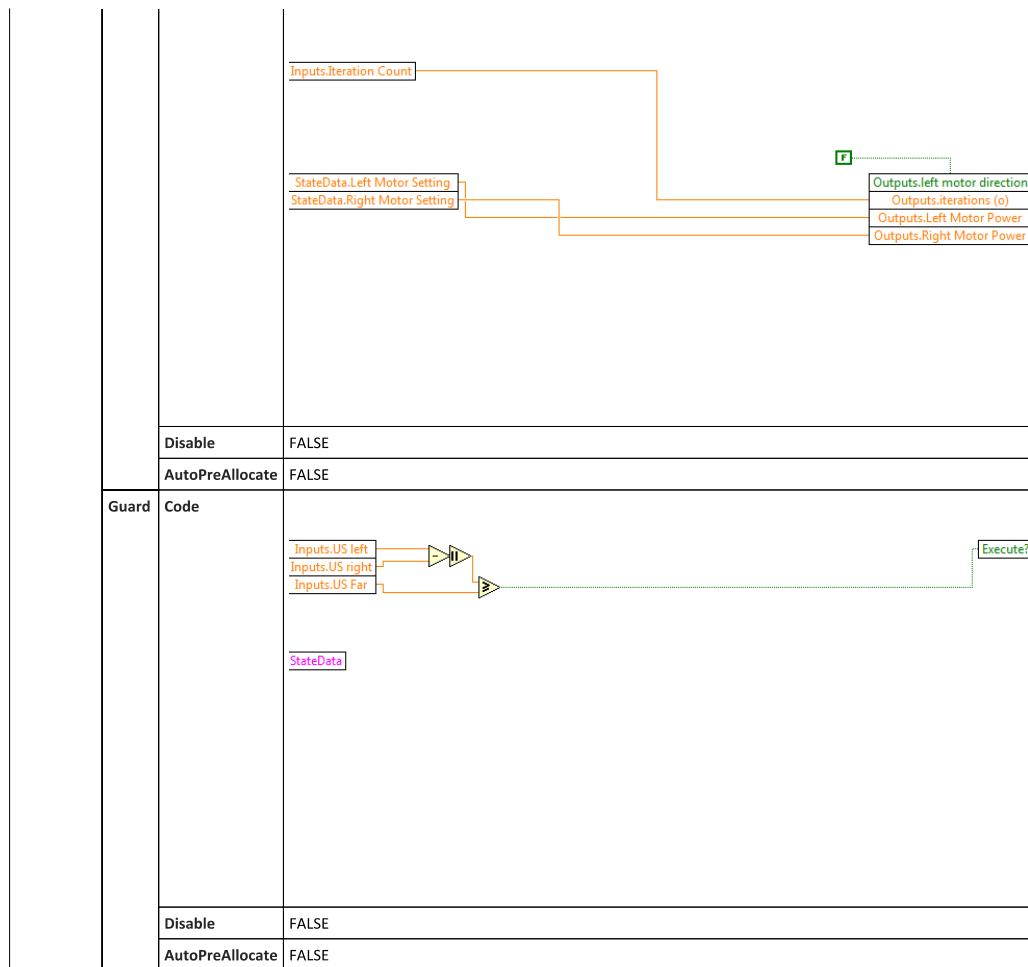
C.1.F.1.T Transition



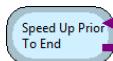
Description		-
Source		Forward to RS
Destination		Spin Right
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.F.1.U Transition**

Description		-
Source		Turn Right
Destination		Forward to RS
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	

**C.1.G Lead Screw Upright**

Description	-
Source	Downhill ->Transition
Destination	Transition ->Downhill
Containing Region	Region

C.1.H Speed Up Prior To End

Description	-
Source	Downhill ->Transition
Destination	Transition ->Downhill
Containing Region	Region

C.1.I Only Left Bumper Hit

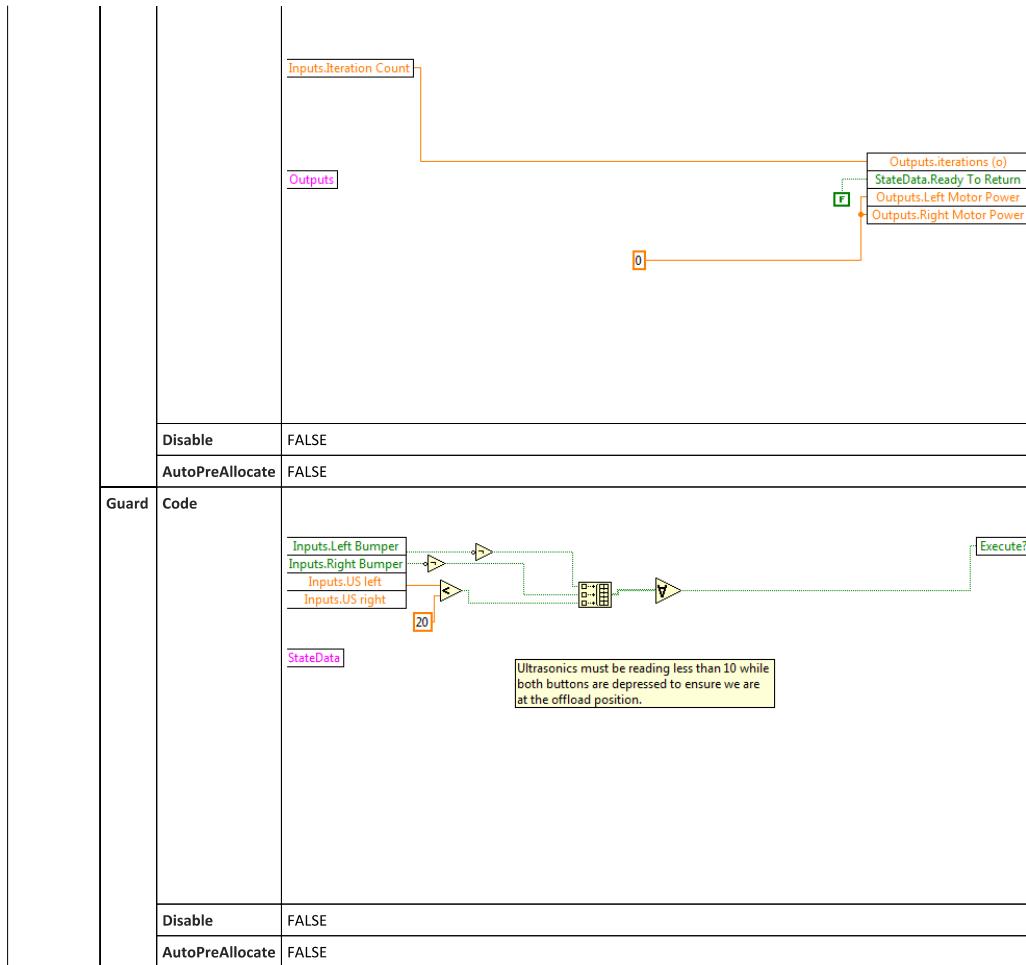
Description	-
Source	Uphill Movement ->Transition
Destination	Transition ->Unload
Containing Region	Region

C.1.J Only Right Bumper Hit

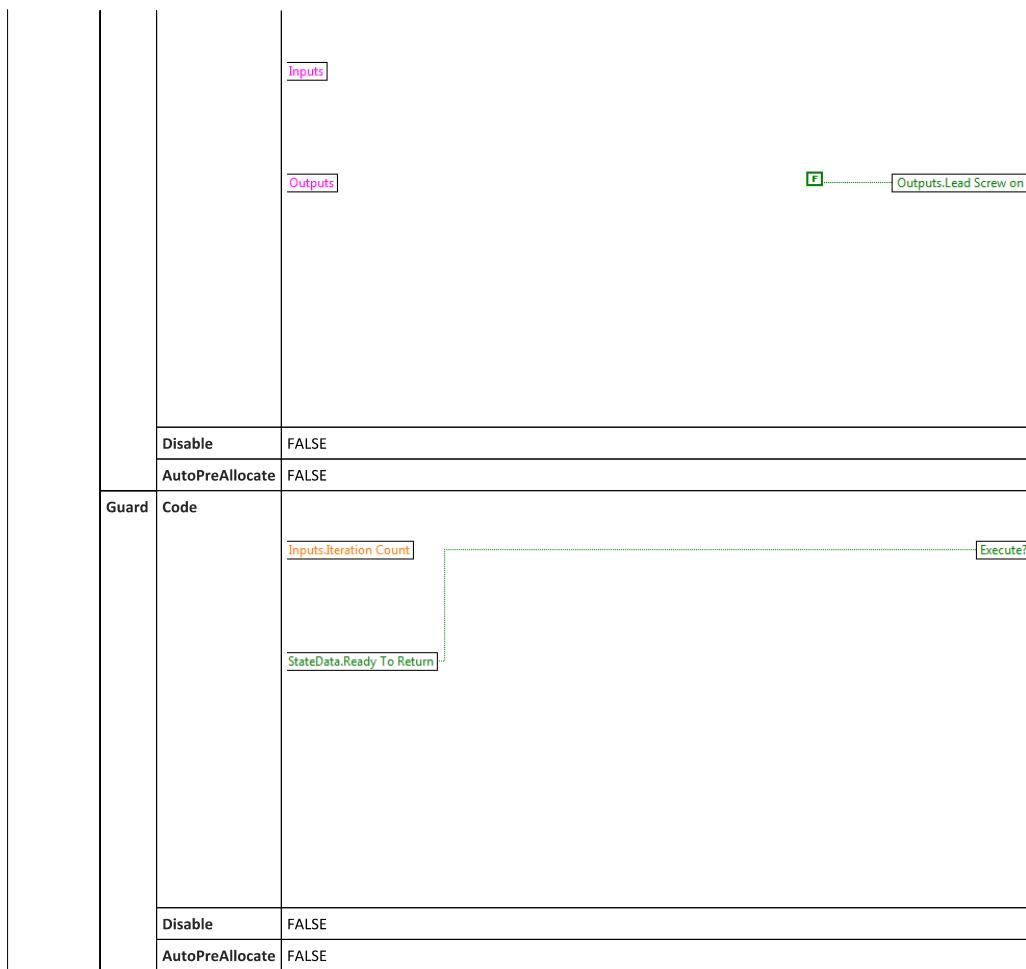
Description	-
Source	Uphill Movement ->Transition
Destination	Transition ->Unload
Containing Region	Region

C.1.K Transition

Description		-
Source		Uphill Movement
Destination		Unload
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**C.1.L Transition**

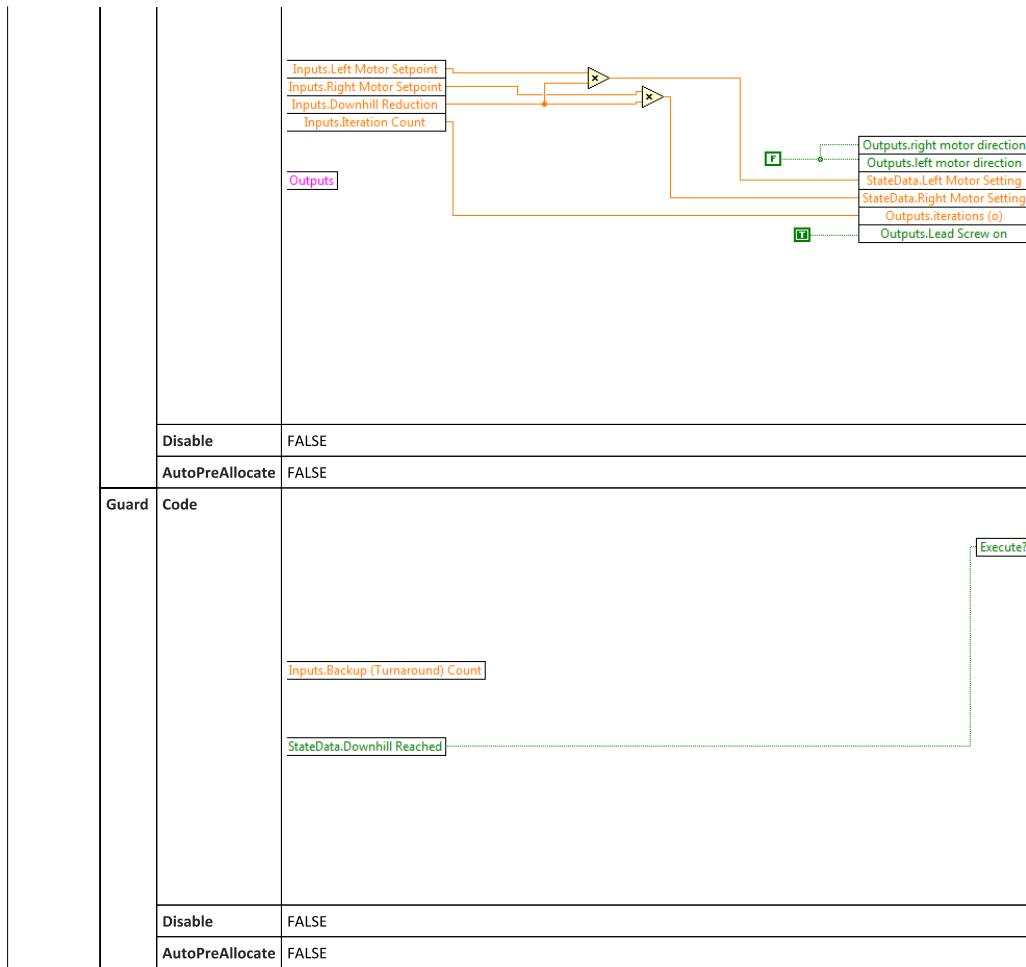
Description		-
Source		Unload
Destination		Turn Around
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code



C.1.M Transition



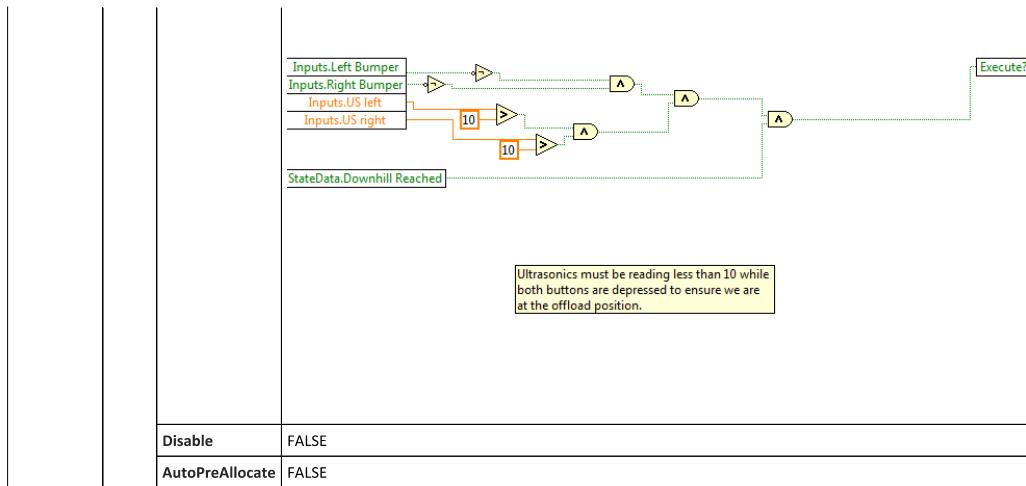
Description		-
Source		Turn Around
Destination		Downhill
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code



C.1.N Transition



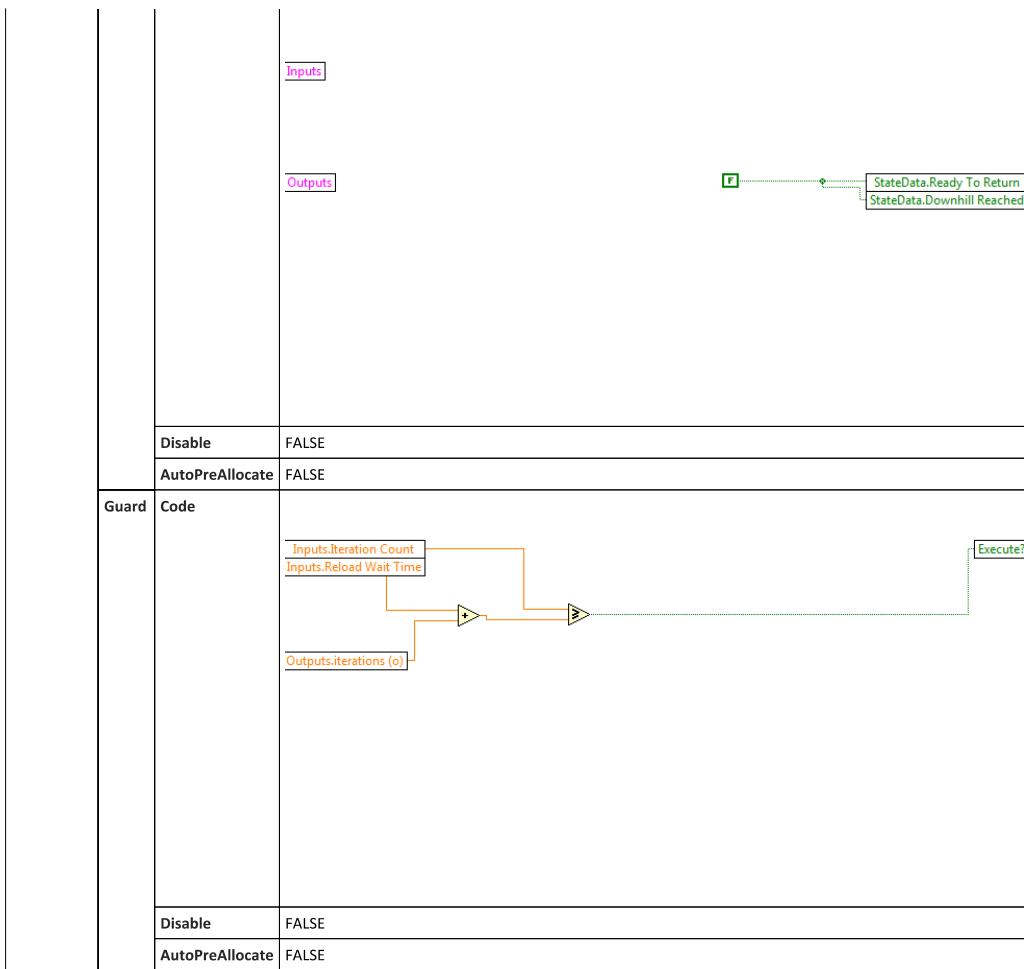
Description	-
Source	Downhill
Destination	Turn Around 2
Containing Region	Region
Transition Code	Description
	Trigger List
	Guard Code

**C.1.O Transition**

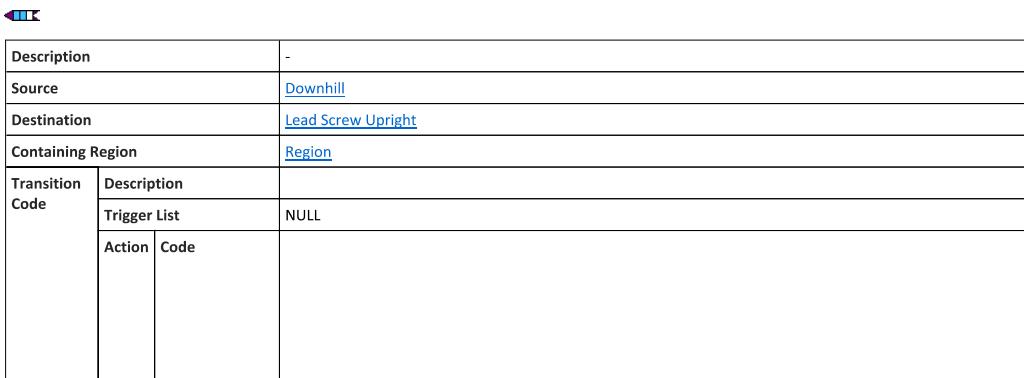
Description	-				
Source	Initial				
Destination	Uphill Movement				
Containing Region	Region				
Transition Code	<table border="1"> <tr> <td>Description</td> <td></td> </tr> <tr> <td>Trigger List</td> <td>{Responds to all triggers}</td> </tr> </table>	Description		Trigger List	{Responds to all triggers}
Description					
Trigger List	{Responds to all triggers}				

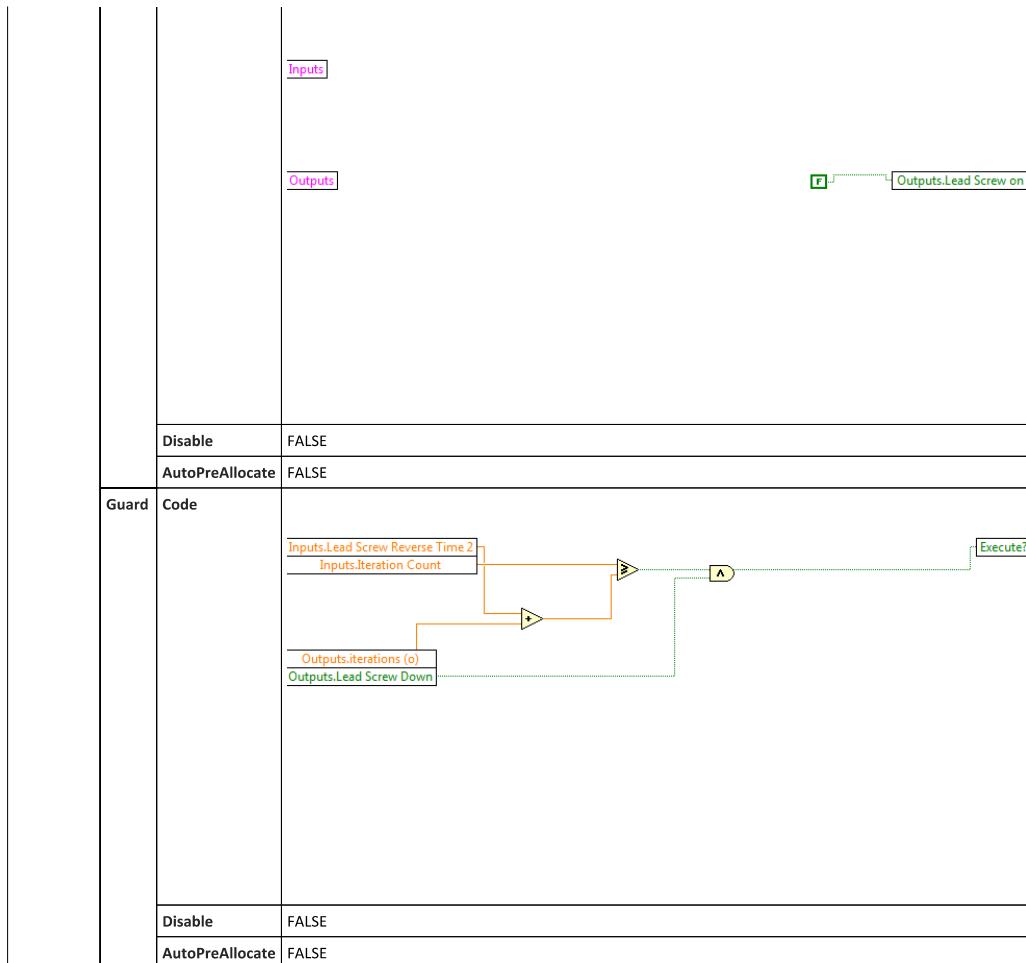
C.1.P Transition

Description	-						
Source	Turn Around 2						
Destination	Uphill Movement						
Containing Region	Region						
Transition Code	<table border="1"> <tr> <td>Description</td> <td></td> </tr> <tr> <td>Trigger List</td> <td>NULL</td> </tr> <tr> <td>Action</td> <td>Code</td> </tr> </table>	Description		Trigger List	NULL	Action	Code
Description							
Trigger List	NULL						
Action	Code						



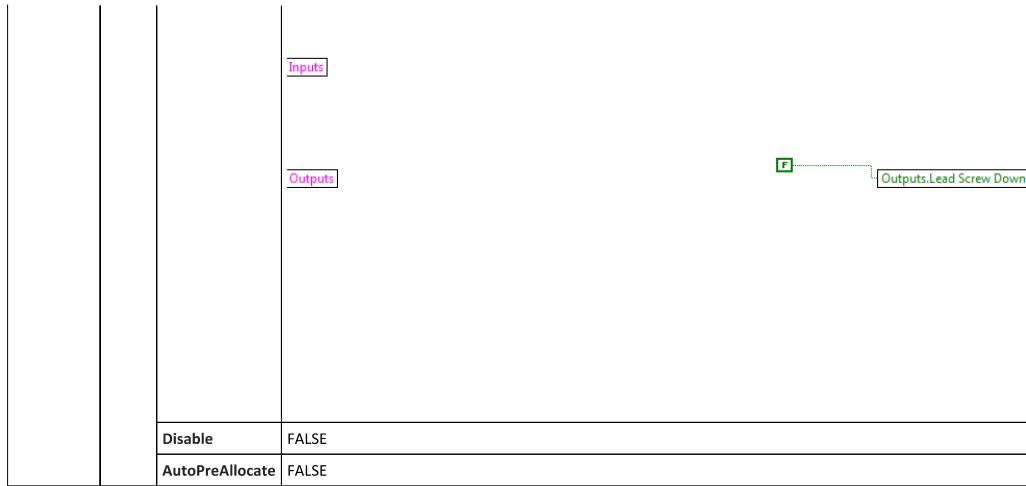
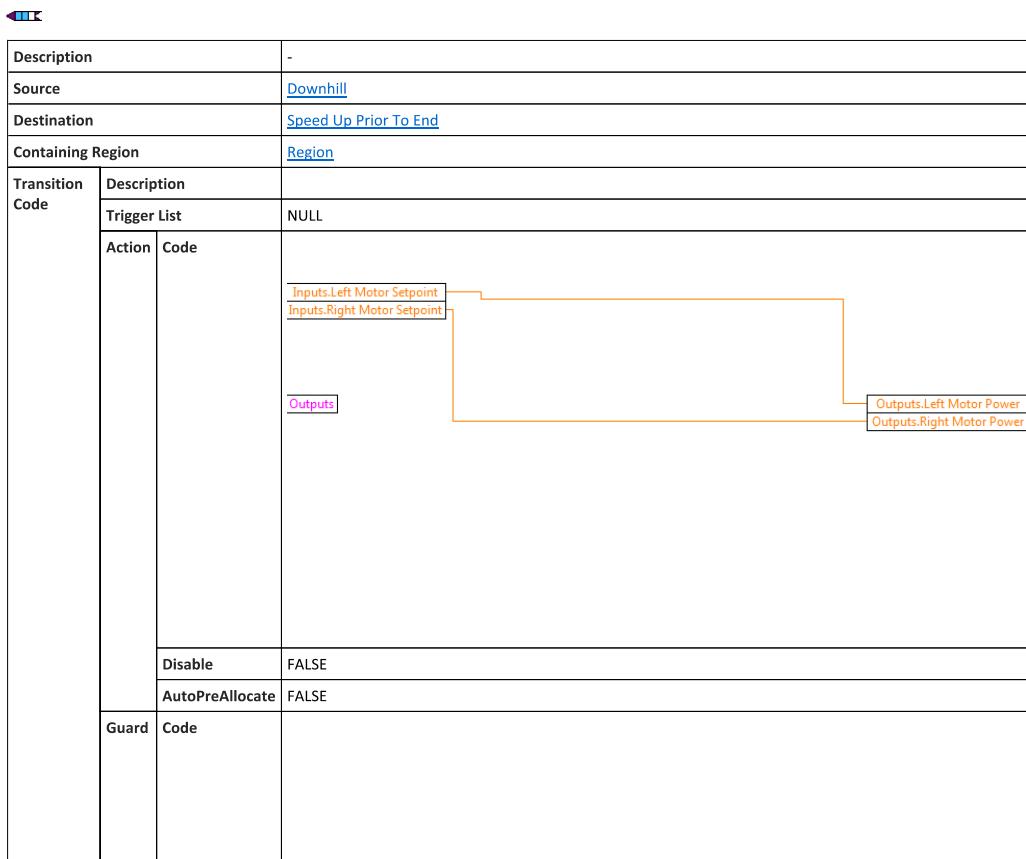
C.1.Q Transition

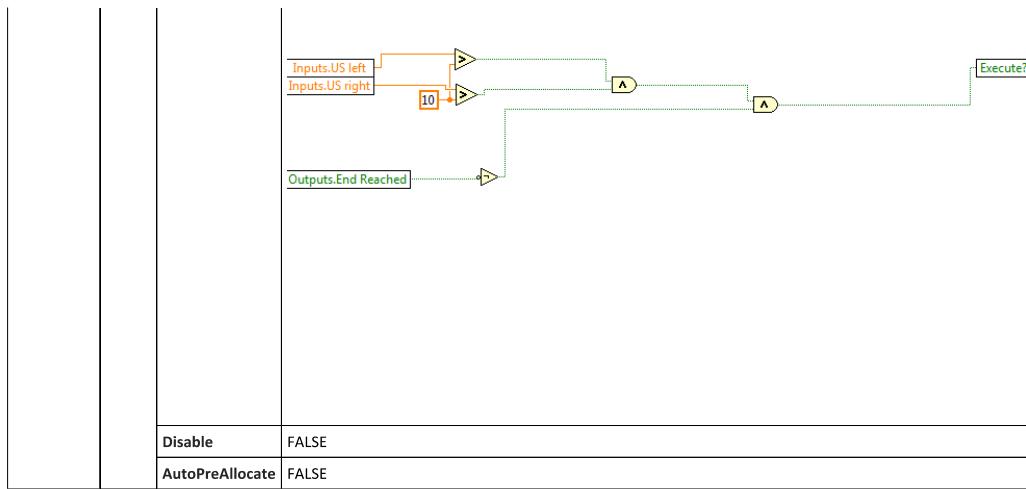




C.1.R Transition

		Description	-
		Source	Lead Screw Upright
		Destination	Downhill
		Containing Region	Region
Transition Code	Description		
	Trigger List		NULL
Action	Code		

**C.1.S Transition**

**C.1.T Transition**

Description		-
Source		Speed Up Prior To End
Destination		Downhill
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code
		 Inputs
		 Outputs
		 Outputs.End Reached
Disable	FALSE	
AutoPreAllocate	FALSE	

C.1.U Transition

Description		-

Source	<u>Uphill Movement</u>	
Destination	<u>Only Left Bumper Hit</u>	
Containing Region	<u>Region</u>	
Transition Code	Description	
	Trigger List	NULL
	Action	Code
		<p>Inputs</p> <p>Outputs</p> <p>800</p> <p>Outputs.Right Motor Power</p>
	Disable	FALSE
	AutoPreAllocate	FALSE
	Guard	Code
		<p>Inputs.Left Bumper</p> <p>Inputs.US left</p> <p>Inputs.US right</p> <p>20</p> <p>StateData</p> <p>A</p> <p>Execute?</p>
	Disable	FALSE
	AutoPreAllocate	FALSE

C.1.V Transition



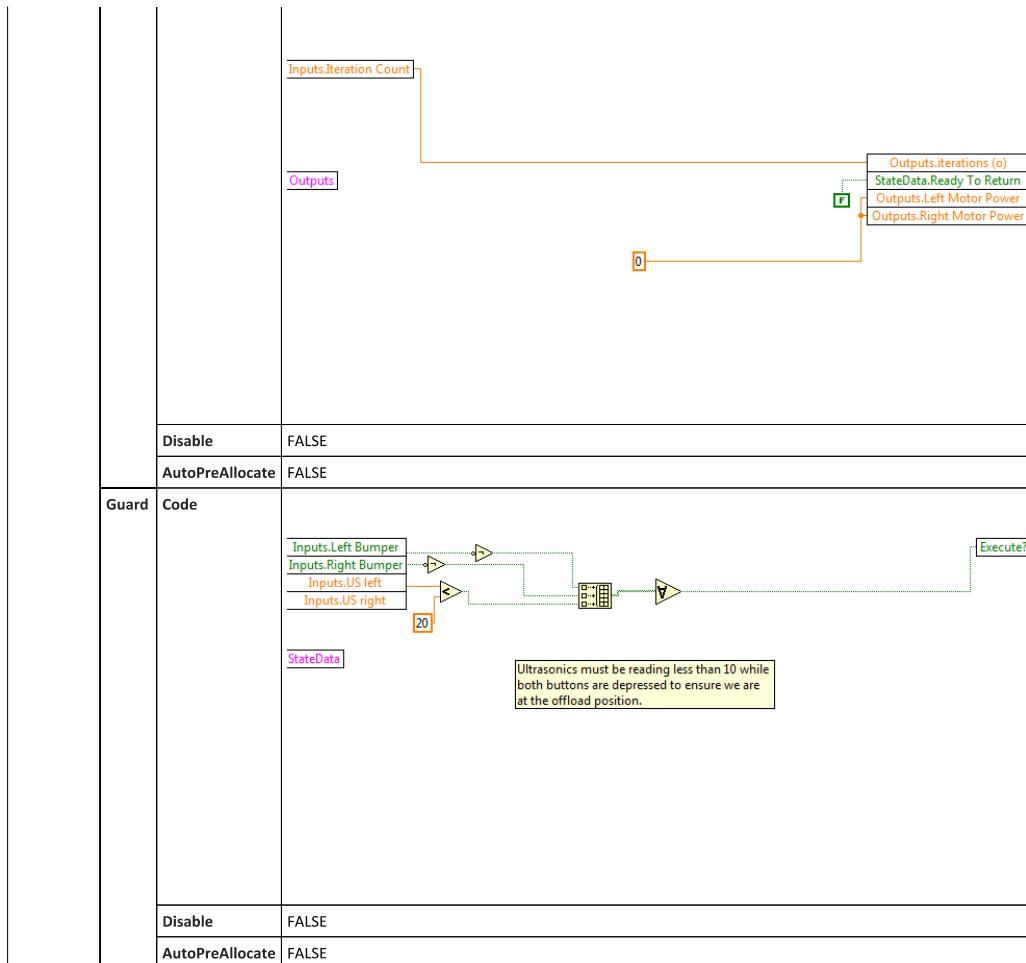
Description	-
Source	<u>Uphill Movement</u>
Destination	<u>Only Right Bumper Hit</u>
Containing Region	<u>Region</u>

Transition Code	Description	
	Trigger List	NULL
Action	Code	<p>Inputs</p> <p>Outputs</p> <p>800</p> <p>Outputs:Left Motor Power</p>
	Disable	FALSE
	AutoPreAllocate	FALSE
Guard	Code	<p>StateData</p>
	Disable	FALSE
	AutoPreAllocate	FALSE

C.1.W Transition



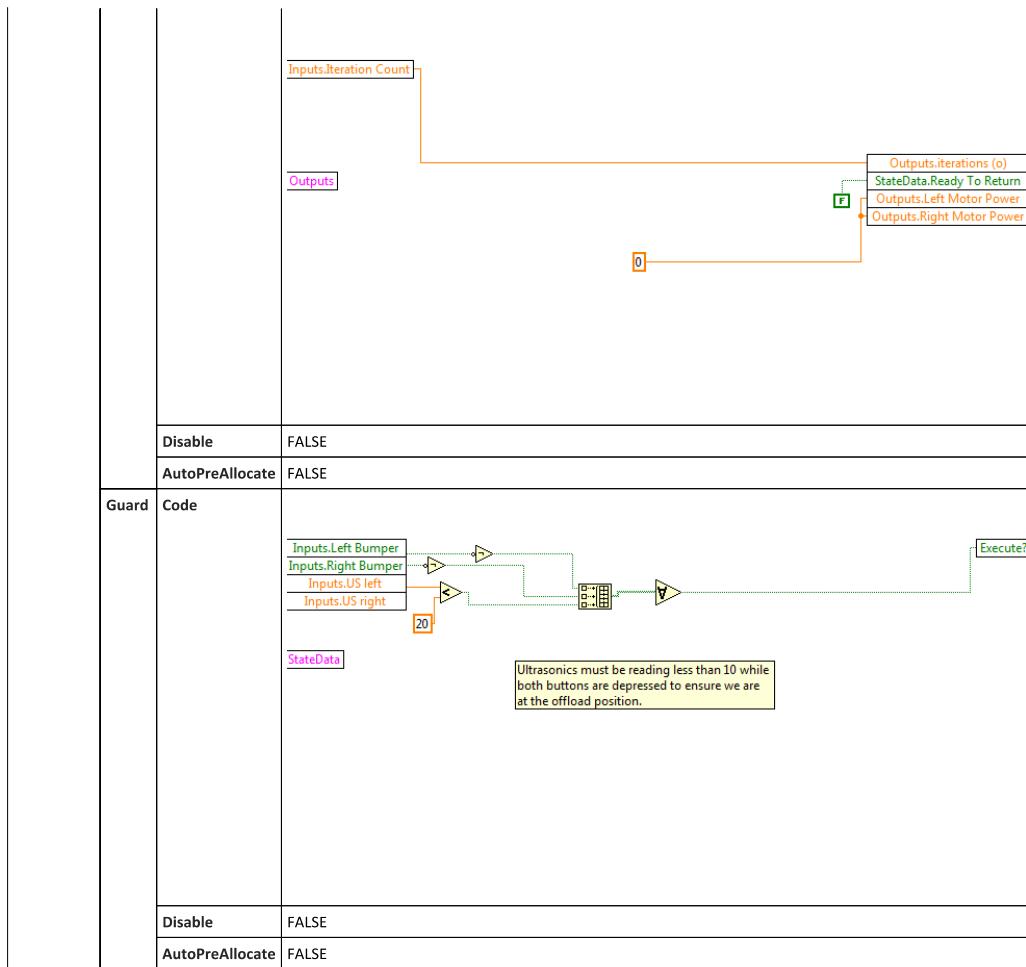
Description		-
Source		Only Left Bumper Hit
Destination		Unload
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
Action	Code	



C.1.X Transition



Description		-
Source		Only Right Bumper Hit
Destination		Unload
Containing Region		Region
Transition Code	Description	
	Trigger List	NULL
	Action	Code

**D Transition**

Description	-
Source	<u>Initial</u>
Destination	<u>off</u>
Containing Region	<u>Main Diagram</u>
Transition Code	Description
	Trigger List {Responds to all triggers}

E Transition

Description	-
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Source		<u>off</u>
Destination		<u>on</u>
Containing Region		Main Diagram
Transition Code	Description	
	Trigger List	NULL
	Action	Code
	Disable	FALSE
	AutoPreAllocate	FALSE
	Guard	Code
		<p>Inputs: System On</p> <p>When On Button (1) is depressed, begin delivery.</p> <p>StateData</p> <pre> graph LR A[Inputs: System On] -- "When On Button (1) is depressed, begin delivery." --> B[Execute?] </pre>
	Disable	FALSE
	AutoPreAllocate	FALSE

F Transition

Description		-
Source		<u>on</u>
Destination		<u>off</u>
Containing Region		Main Diagram
Transition Code	Description	

	Trigger List	NULL
Action	Code	<p>Inputs</p> <p>Outputs</p> <pre> graph LR A[Inputs.motors on] --- B[AND] C[Inputs.lead screw on] --- B B --- D[OR] E[Inputs.left motor direction] --- D F[Inputs.right motor direction] --- D D --- G[NOT] G --- H[AND] I[StateData.ready to return] --- H J[StateData.downhill reached] --- H H --- K[Outputs.end reached] </pre>
Disable	FALSE	
AutoPreAllocate	FALSE	
Guard	Code	<p>Inputs.System Off</p> <p>StateData</p> <pre> graph LR A[Inputs.System Off] --- B[AND] B --- C[StateData] C --- D[Execute?] </pre>
Disable	FALSE	
AutoPreAllocate	FALSE	