3.3.1 Buckling Limit State

Loading Condition $\eta := 1.0$

Material Information

E := 206000v := 0.3 $\sigma o := 355$

Panel Information

1:=3628

s := 1340 t := 19.0

C1 := 1.1 C2 := 1.2

 $\sigma x := 12.7 \quad \sigma y := 70.6 \quad \tau := 103.0$

$$\left(\frac{\sigma x}{\eta \cdot \sigma c x}\right)^2 + \left(\frac{\sigma y}{\eta \cdot \sigma c y}\right)^2 + \left(\frac{\tau}{\eta \cdot \tau c}\right)^2 \le 1.0$$

Critical Buckling Stress: σcx

$\kappa := 1.0$ Pr := 0.6 $ksx := C1 \cdot if (\kappa \ge 0) \land (\kappa \le 1)$ $\frac{\kappa+1}{\kappa+1}$ else if $(\kappa \ge -1) \wedge (\kappa < 0)$

oEx
else
$$\sigma o \cdot \left(1 - Pr \cdot (1 - Pr) \cdot \frac{\sigma o}{\sigma Ex}\right)$$

 $\sigma cx := \text{if } \sigma Ex \leq Pr \cdot \sigma o$

ksx = 4.4

 $\sigma Ex = 164.7$

 $\sigma cx = 164.7$

Critical Buckling Stress: σcx

$$\begin{aligned} & \kappa := 1.0 \\ & Pr := 0.6 \\ & \alpha := \frac{1}{s} \\ & ksy := C2 \cdot \text{if} \left(\left[\kappa < \frac{1}{3} \right] \wedge \left(\alpha \ge 1 \right) \right) \wedge \left(\alpha \le 2 \right) \\ & \left(1.0875 \cdot \left(1 + \frac{1}{\alpha^2} \right)^2 - 18 \cdot \frac{1}{\alpha^2} \right) \cdot \left(1 + \kappa \right) + 24 \cdot \frac{1}{\alpha^2} \\ & \text{else} \\ & \text{if} \left(\left[\kappa < \frac{1}{3} \right] \wedge \left(\alpha > 2 \right) \right) \\ & \left(1.0875 \cdot \left(1 + \frac{1}{\alpha^2} \right)^2 - 9 \cdot \frac{1}{\alpha} \right) \cdot \left(1 + \kappa \right) + 12 \cdot \frac{1}{\alpha} \\ & \text{else} \\ & \left(1 + \frac{1}{\alpha^2} \right)^2 \cdot \left(1.675 - 0.675 \cdot \kappa \right) \end{aligned}$$

$$& \sigma Ey := ksy \cdot \frac{\mathbf{\pi}^2 \cdot E}{12 \cdot \left(1 - v^2 \right)} \cdot \left(\frac{t}{s} \right)^2$$

$$& \sigma Cy := \text{if} \quad \sigma Ey \le Pr \cdot \sigma O$$

$$& \sigma Ey \\ & \text{else}$$

$$& \sigma O \cdot \left(1 - Pr \cdot \left(1 - Pr \right) \cdot \frac{\sigma O}{\sigma Ey} \right) \end{aligned}$$

$$ksy = 1.55$$

$$\sigma Ey = 58.01$$

$$\sigma cy = 58.01$$

Critical Buckling Stress : т

$$\tau o := \frac{1}{\sqrt{3}} \cdot \sigma o$$

$$Pr := 0.6$$

$$ks := \left(4.0 \cdot \left(\frac{s}{I}\right)^2 + 5.34\right) \cdot C1$$

$$\tau E := ks \cdot \frac{\pi^2 \cdot E}{12 \cdot \left(1 - v^2\right)} \cdot \left(\frac{t}{s}\right)^2$$

$$\tau c := \text{if } \tau E \le Pr \cdot \tau o$$

$$\tau E$$

$$\text{else}$$

$$\tau o \cdot \left(1 - Pr \cdot \left(1 - Pr\right) \cdot \frac{\tau o}{\tau E}\right)$$

$$ks = 6.474$$

$$\tau E = 242.343$$

$$\tau c = 163.357$$

$$\left(\frac{\sigma x}{\eta \cdot \sigma c x}\right)^2 + \left(\frac{\sigma y}{\eta \cdot \sigma c y}\right)^2 + \left(\frac{\tau}{\eta \cdot \tau c}\right)^2 = 1.885$$

3.3.3 Ultimate Strength under Combined in-Plane Stresses

$$\left(\frac{\sigma x}{\eta \cdot \sigma U x}\right)^2 + \varphi\left(\frac{\sigma x}{\eta \sigma U x}\right) \cdot \left(\frac{\sigma y}{\eta \cdot \sigma U y}\right) + \left(\frac{\sigma y}{\eta \cdot \sigma U y}\right)^2 + \left(\frac{\tau}{\eta \cdot \tau U}\right)^2 \leq 1.0$$

slenderness ratio

$$\beta := \frac{s}{t} \cdot \sqrt{\frac{\sigma o}{E}} = 2.928$$

coefficient to reflect interaction between longitudinal and transverse stresses $\varphi := 1.0 - \frac{\beta}{2} = -0.464$

$$\sigma Ux = 201.093$$

$$\sigma Uy = 102.19$$

$$\tau U = 174.202$$

$$\left(\frac{\sigma x}{\eta \cdot \sigma U x}\right)^{2} + \varphi \cdot \left(\frac{\sigma x}{\eta \cdot \sigma U x}\right) \cdot \left(\frac{\sigma y}{\eta \cdot \sigma U y}\right) + \left(\frac{\sigma y}{\eta \cdot \sigma U y}\right)^{2} + \left(\frac{\tau}{\eta \cdot \tau U}\right)^{2} = 0.811$$