

# An AMG preconditioner for moving mesh finite element method

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**Abstract.** In this paper, we apply an AMG preconditioner to solve the unsteady Navier-Stokes equations with moving mesh finite element method.  $4P1 - P1$  element pair is selected, which based on the data structure of hierarchy geometry tree. We choose two-layer nested meshes that velocity mesh and pressure mesh. AMG preconditioners are designed for PDE solver and divergence-interpolation in moving mesh strategy. Numerical experiments show the efficiency of the AMG preconditioner for moving mesh finite element.

**AMS subject classifications:** 65M10, 78A48

**Key words:** Navier-Stokes, algebraic multigrid precondition, moving mesh.

## 1. Introduction

The incompressible Navier-Stokes equations in primitive variables are

$$\begin{aligned} \partial_t \vec{u} - \nu \nabla^2 \vec{u} + (\vec{u} \cdot \nabla) \vec{u} + \nabla p &= \vec{f}, \\ \nabla \cdot \vec{u} &= 0, \end{aligned} \quad (1.1)$$

with initial and boundary conditions on  $\partial\Omega = \partial\Omega_D \cup \partial\Omega_N$ :

$$\begin{aligned} \vec{u} &= \vec{w}, & \text{on } \partial\Omega_D \times [0, T] \\ \nu \frac{\partial \vec{u}}{\partial n} - p &= \vec{0}, & \text{on } \partial\Omega_N \times [0, T], \\ \vec{u}|_{t=0} &= \vec{u}_0, & \text{in } \Omega. \end{aligned} \quad (1.2)$$

where  $\Omega \in \mathcal{R}^d$ , ( $d = 2, 3$ ) is computational domain,  $[0, T]$  is the time interval,  $\vec{u}$  is velocity and scalar  $p$  is pressure,  $\vec{n}$  denotes outward normal direction of  $\partial\Omega$ ,  $\nu > 0$  is the constant kinematic viscosity.

We solve (1.1) and (1.2) by moving mesh finite element methods based on [1] and [2]. In the past, some moving mesh methods have been introduced. Winslow [3] proposed

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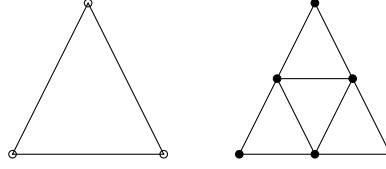


Figure 1: Left: pressure  $p$  element,  $\circ$  for degrees of  $p$ ; right: four velocity  $v$  elements,  $\bullet$  for degrees of  $v$ .

solving elliptic PDEs using moving mesh. As an extension of Winslow's work, Dvinsky [4] pointed out that harmonic function theory could be used for generating mesh. Motivated by Dvinsky's work, Li, Tang and Zhang [1] proposed a moving mesh finite element strategy based upon harmonic mapping. The authors in [5] extended the moving strategy to solve the incompressible Navier-Stokes equations in primitive variables. The author designed a divergence-free interpolation in moving strategy by solving a linearized Navier-Stokes-type equations. In [6],  $4P1 - P1$  element pair is applied to solve incompressible Navier-Stokes flow with moving mesh finite element method based on the work of [5]. This pair has same mesh structure as  $P1isoP2P1$  element, which is naturally LBB stable see [7]. Four velocity elements can be obtained by refining the pressure element one time see Figure 1. Linear velocity basis functions of  $4P1 - P1$  are all locally in the same velocity element, whereas  $P1isoP2P1$  not, see [6] for detail.

As we known, spacial discretization of Navier-Stokes system with LBB-stable  $4P1 - P1$  element pair leads to a saddle point problem. There are a lot works on saddle point problems by developing preconditioners for Krylov subspace method, such as block preconditioner and multigrid strategy. Readers can refer to [8] for detail. Two-grid method was introduced to solve Navier-Stokes equations, see ([9], [10] and [11]) for details. Many works ([12], [13], [14], [15]) introduce a variety of block preconditioners, whose main issue are finding a good approximation of schur complement. Also there are other precondition methods, for instance ([16], [17]). The authors in ([18] [19]) propose an efficient AMG preconditioner for Krylov solver to solve Navier-Stokes equations. However, efficient precondition methods for saddle point problems are nearly based on uniform mesh (although the stretched mesh case is considered in [17]).

In this work, we apply an AMG preconditioner to moving mesh finite element for solving systems (1.1) and (1.2) based on the work of [20]. Also AMG precondition strategy is designed for divergence-free interpolation in moving mesh method. Efficiency of the AMG preconditioner is analyzed through several numerical experiments.

The layout of the paper is arranged as follows. In section 2, we use  $4P1 - P1$  element to approximate the governing equations. Next, the AMG preconditioner for Navier-Stokes equations is shown. In Section 4, we give the moving mesh strategy briefly. Then we present numerical experiments in section 5. Finally, we give the conclusions in this section.

## 2. Data structure and weak formulation

At time level discretization, we divide the time interval  $[0, T]$  into  $N$  steps with  $\{t_i\}_{i=1}^N$ . Let  $\vec{u}^j$  and  $p^j$  be the discrete approximation to  $\vec{u}(\cdot, t_j)$  and  $p(\cdot, t_j)$ . For simplicity, we choose linear backward Euler scheme that linearizing the nonlinear term  $(\vec{u}^{n+1} \cdot \nabla) \vec{u}^{n+1}$  with  $(\vec{u}^n \cdot \nabla) \vec{u}^{n+1}$ .

In this work, we adopt finite element pair  $4P1 - P1$ , which based on two different triangular meshes and two different finite element spaces. By using the hierarchy geometry tree ([21]) structure, velocity mesh can be obtained via global refining pressure mesh one time see Figure 2.

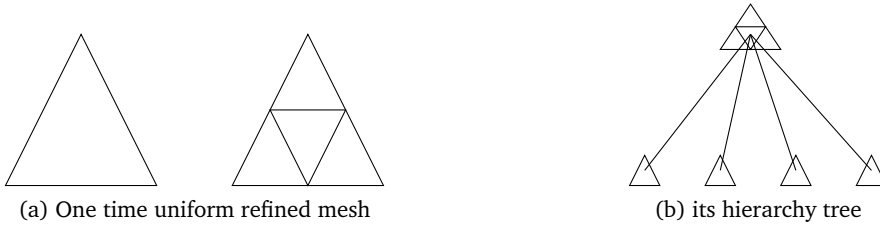


Figure 2: Hierarchy tree structure

The 1 – 1 index between velocity elements and pressure elements can be obtained without difficulties with the hierarchy geometry tree structure. Interested author see [6] for details. First some notation are denoted as follows.  $\mathcal{T}_h$  is the grid triangulation division for velocity mesh with mesh size  $h = \max_{T \in \mathcal{T}_h} \text{diam}(T)$ , while  $\mathcal{T}_H (H = 2h)$  for pressure mesh.  $X_h \subset (H_0^1(\Omega)^2)$  and  $P_H \subset \mathcal{L}^2(\Omega)$  are finite-dimensional approximation spaces. Then the full discretization is the following: given  $(\vec{u}_h^n, p_H^n)$  at time  $t_n$ , to compute  $(\vec{u}_h^{n+1}, p_H^{n+1})$  via

$$\begin{aligned} \frac{1}{dt}(\vec{u}_h^{n+1}, \vec{v}_h) + \nu(\nabla \vec{u}_h^{n+1}, \nabla \vec{v}_h) + (\vec{u}_h^n \cdot \nabla \vec{u}_h^{n+1}, \vec{v}_h) - (p_H^{n+1}, \nabla \vec{v}_h) &= \frac{1}{dt}(\vec{u}_h^n, \vec{v}) \\ (\nabla \cdot \vec{u}_h^{n+1}, q_H) &= 0. \end{aligned} \quad (2.1)$$

for all  $(\vec{v}_h, q_H) \in \mathcal{X}_h \times P_H$ .

## 3. Fast krylov solver with AMG precondition strategy

Let  $(\{\phi_j\}_{j=1}^n, 0)^T$  and  $(0, \{\psi_k\}_{k=1}^m)^T$  be linear basis functions for velocity space  $X_h$ . Meanwhile,  $\{\psi_k\}_{k=1}^m$  denotes linear basis functions for pressure space  $P_H$ . Then components of velocity solutions  $\vec{u}_h^{n+1} = (u_h^{x,n+1}, u_h^{y,n+1})^T$  and pressure solution  $p_H^{n+1}$  at  $t = t_{n+1}$  can be written as

$$u_h^{x,n+1} = \sum_{j=1}^{n_u} \alpha_j^{x,n+1} \phi_j, \quad u_h^{y,n+1} = \sum_{j=1}^{n_u} \alpha_j^{y,n+1} \phi_j, \quad p_H^{n+1} = \sum_{k=1}^{n_p} \alpha_k^{p,n+1} \psi_k. \quad (3.1)$$

substituting (3.1) into weak form (2.1), a linear system can be obtained

$$\begin{bmatrix} \frac{1}{dt}M + \nu A + N & 0 & B_x^T \\ 0 & \frac{1}{dt}M + \nu A + N & B_y^T \\ B_x & B_y & 0 \end{bmatrix} \begin{bmatrix} \alpha^{x,n+1} \\ \alpha^{y,n+1} \\ \alpha^{p,n+1} \end{bmatrix} = \begin{bmatrix} f_x \\ f_y \\ 0 \end{bmatrix}, \quad (3.2)$$

Notice that divergence matrix  $B = [B_x, B_y]$  is

$$B_x := [B_x]_{kj} = - \left( \psi_k, \frac{\partial \phi_j}{\partial x} \right), k = 1, \dots, n_p, j = 1, \dots, n_u, \quad (3.3)$$

$$B_y := [B_y]_{kj} = - \left( \psi_k, \frac{\partial \phi_j}{\partial y} \right), k = 1, \dots, n_p, j = 1, \dots, n_u. \quad (3.4)$$

Assembling of matrix  $B$  is a non-trivial process due to the basis functions of velocity elements and pressure elements are on different mesh. According to the 1 – 1 index between velocity elements and pressure elements mentioned above, we can just use local  $P1$  element of both velocity and pressure elements to assemble  $B$ .  $B^T$  is in the same way.

We denote  $F_v^{n+1} = \frac{1}{dt}M + \nu A + N$ , where

$$M := [M]_{ij} = (\phi_i, \phi_j), \quad i, j = 1, \dots, n_u, \quad (3.5)$$

$$A := [A]_{ij} = (\nabla \phi_i, \nabla \phi_j), \quad i, j = 1, \dots, n_u, \quad (3.6)$$

$$N := [N]_{ij} = (\bar{u}_h^n, \nabla \phi_i, \phi_j), \quad i, j = 1, \dots, n_u. \quad (3.7)$$

To solve linear system (3.2) efficiently, we use preconditioned GMRES as solver. The block triangular preconditioner  $\mathcal{P}$  discussed in [20] which is defined as following

$$\mathcal{P} = \begin{pmatrix} F & 0 & B_x^T \\ 0 & F & B_y^T \\ 0 & 0 & S \end{pmatrix} \quad (3.8)$$

where  $S = B_x F^{-1} B_x^T + B_y F^{-1} B_y^T$  is the schur complement matrix. The action of  $\mathcal{P}^{-1}$  is divided into two steps: first, solve schur complement system, second, solve two scalar systems associated with  $F$ . It is costly to directly solve schur complement system. So in practical computation, the PCD preconditioner discussed in [20] is used to approximate schur complement matrix  $S$ . PCD preconditioner is denoted as  $S_* = A_p F_p^{-1} Q_p$ , where  $A_p, F_p$  and  $Q_p$  are all on the pressure space.  $Q_p$  is mass matrix,  $A_p$  is pressure diffusion matrix and  $F_p$  is convection diffusion matrix denoted as

$$F_p := [F_p]_{ij} = \nu(\nabla \psi_i, \nabla \psi_j) + (\bar{u}_h^n \cdot \nabla \psi_i, \psi_j), \quad i, j = 1, \dots, n_p, \quad (3.9)$$

$$A_p := [A_p]_{ij} = (\nabla \psi_i, \nabla \psi_j) \quad i, j = 1, \dots, n_p. \quad (3.10)$$

Let  $W_p^n := [W_p^n]_{ij} = (\bar{u}_h^n \cdot \nabla \psi_i, \psi_j)$ ,  $i, j = 1, \dots, n_p$ , then  $F_p$  can be rewritten as  $F_p = \nu A_p + W_p^n$ . We implement PCD preconditioning by

$$S_*^{-1} \approx Q_p^{-1} F_p A_p^{-1}. \quad (3.11)$$

Exact PCD preconditioning operator is denoted as

$$\mathcal{M}^{-1} = \begin{pmatrix} F^{-1} & 0 & B_x^T \\ 0 & F^{-1} & B_y^T \\ 0 & 0 & S_*^{-1} \end{pmatrix} \quad (3.12)$$

We will explain the  $V^d = \mathcal{M}^{-1}V^s$  in two steps, where  $V^d = (V_x^d, V_y^d, V_p^d)^T$ ,  $V^s = (V_x^s, V_y^s, V_p^s)^T$ . Firstly, we solve

$$V_p^d = S_*^{-1}V_p^s = Q_p^{-1}F_pA_p^{-1}V_p^s. \quad (3.13)$$

It contains two possion problems  $Q_p^{-1}$  and  $A_p^{-1}$ , so we can use just an AMG solver for possion equation to solve them. Secondly, we use AMG solver to solve

$$\begin{aligned} V_x^d &= F^{-1}(V_x^s - B_x^TV_p^d), \\ V_y^d &= F^{-1}(V_y^s - B_y^TV_p^d). \end{aligned} \quad (3.14)$$

Combine (3.13) and (3.14), then  $V^d$  is obtained.

In practical computation, we use a fixed number of AMG iterations (usually one or two) for matrix  $F$ ,  $F_p$ ,  $Q_p$ , and  $A_p$  to replace accurate solving, which refers to iterated PCD preconditioning. The AMG solver is based on the AFPack(an adaptive finite element package), which can be obtained from <http://dsec.pku.edu.cn/~rli>. The efficiency of PCD preconditioning is shown in ([20], Section 10) and [22] for bouyancy driven flow problem. In our experiements,  $F_p$  in (3.10) is not so efficient as getting rid of  $v$  in (3.10). If without explanation, we refer  $F_p = A_p + W_p^n$  in this paper. We compare the efficiency of two choices of  $F_p$  in numerical test below. We adopt the method in [15] to deal with matrixes  $F_p$  and  $A_p$  on Neumann boundary for improving efficiency.  $F_p$  should satisfy condition

$$v \frac{\partial p_h}{\partial n} + (\vec{w}_h \cdot \vec{n})p_h = 0. \quad (3.15)$$

on boundary  $\partial\Omega$ . We know that for cavity flow, (3.15) will become  $\frac{\partial p_h}{\partial n} = 0$ , which means do-nothing for  $F_p$  on boundary  $\Omega$ . In this work, we apply the PCD preconditioning strategy to moving mesh finite element method to efficiently solve system (3.2). The moving strategy will be shown in next section.

## 4. Moving mesh strategy

### 4.1. Moving mesh framework

We refer the moving strategy to [5]. In the following, we briefly introduce the moving mesh method. At time  $t = t_n$ , we obtain numerical solutions  $\vec{u}_h^{(n)}, p_H^{(n)}$  on old mesh  $\mathcal{T}_h^n$ . We follow the framework in [5] to implement divergence-free interpolation of solutions on  $\mathcal{T}_h^n$  to new mesh  $\mathcal{T}_h^{(n+1)}$ . Briefly speaking, the moving mesh strategy mainly contains four steps as follows.

step 1 Obtain monitor function. It is very important to choose an appropriate monitor function for adaptive scheme. Let  $m = 1/G$ , where  $G$  is the monitor function. As illustrated in [5], there are some common choices of  $G$ . One based on vorticity is

$$G = \sqrt{1 + \alpha|\omega|^\beta}. \quad (4.1)$$

where  $\omega = \nabla \times \vec{u}$ ,  $\alpha, \beta$  are positive constants. In this work,  $\beta = 2$  performs well, while  $\alpha$  is user defined according to different problems.

step 2 Get a new logical mesh. Solve elliptic equation

$$\begin{aligned} \nabla_{\vec{x}}(m\nabla_{\vec{x}}\vec{\xi}) &= 0, \\ \vec{\xi}|_{\partial\Omega} &= \vec{\xi}_b. \end{aligned} \quad (4.2)$$

where  $m$  is given in step 1. Then a new logical mesh  $\mathcal{T}_c^*$  with  $\mathcal{A}^*$  as nodes is obtained.

step 3 Achieve mesh move direction in physical domain. First, After Step 1 and Step 2, we obtain a new logical mesh  $\mathcal{T}_c^*$  (with  $\mathcal{A}^*$  as node). Then we can get the difference between  $\mathcal{T}_c^*$  and initial logical mesh  $\mathcal{T}_c^0$  (with nodes  $\mathcal{A}^0$ ):

$$\delta\mathcal{A} = \mathcal{A}^0 - \mathcal{A}^*. \quad (4.3)$$

The displacement  $\delta X_i$  in physical domain can be obtained with  $\delta\mathcal{A}$ . Moreover a positive parameter  $\mu$  is multiplied to the displacement  $\delta X_i$  in updating old mesh in physical domain to a new one:

$$X_i^{(n+1)} = X_i^{(n)} + \mu\delta X_i. \quad (4.4)$$

Interested readers can see [23] for mesh redistribution in detail.

step 4 Preserve divergence-free interpolation. It is necessary to keep divergence-free in the interpolation when solving incompressible flow with moving mesh finite element method. In [5], solution re-distribution on the new mesh  $\mathcal{T}^{(n+1)}$  is achieved via solving a linearad inviscid Navier-Stokes-type system as following

$$\begin{aligned} \frac{\partial \vec{u}}{\partial \tau} - \nabla_{\vec{x}} \vec{u} \cdot \delta \vec{x} &= -\nabla \hat{p}, \\ \nabla_{\vec{x}} \cdot \vec{u} &= 0. \end{aligned} \quad (4.5)$$

where  $\delta \vec{x} := x^{\text{old}} - x^{\text{new}}$  and  $x^{\text{old}}, x^{\text{new}}$  are two sets of coordinates in physical domain.  $\tau$  is a virtual time variable and often chosen as 1.0, because of the convection speed  $\delta \vec{x}$  is relatively small. Here  $\hat{p}$  is a temporary variable distinguished from the pressure variable in (1.1).

Weak form of (4.5) is : find  $(\vec{u}_h, \hat{p}_H) \in X_E^h \times P^H$  such that

$$\begin{aligned} (\partial_\tau \vec{u}_h - \nabla_{\vec{x}} \vec{u}_h \cdot \delta \vec{x}, \vec{v}_h) &= (\hat{p}_H, \nabla \vec{v}_h), \quad \forall \vec{v}_h \in X_E^h, \\ (\nabla_{\vec{x}} \cdot \vec{u}, q_H) &= 0, \quad \forall q_H \in P^H. \end{aligned} \quad (4.6)$$

In this work, we use explicit scheme to (4.6) for time discretization:

$$\begin{aligned} \left( \frac{\vec{u}_{h,*}^{(n)} - \vec{u}_h^{(n)}}{\delta t}, \vec{v}_h \right) + \left( \delta \vec{x} \cdot \nabla \vec{u}_h^{(n)}, \vec{v}_h \right) &= \left( \hat{p}_{H,*}^{(n)}, \nabla \vec{v}_h \right), \quad \forall \vec{v}_h \in X_E^h. \\ (\nabla \cdot \vec{u}_{h,*}^n, q_H) &= 0, \quad \forall q_H \in P^H. \end{aligned} \quad (4.7)$$

where  $\vec{u}_h^{(n)}$  and  $p_H^{(n)}$  are the numerical solutions of (1.1) at  $t = t_n$  using the mesh at  $t_n$ .  $\vec{u}_{h,*}^{(n)}$  and  $p_{h,*}^{(n)}$  are the intermediate updated solutions at  $t_n$  on the new mesh  $\mathcal{T}^{(n+1)}$ .

#### 4.2. AMG preconditioning strategy for (4.7) in solution re-distribution

(4.7) will bring out a linear system, whose coefficient matrix  $\mathcal{M}^p$  can be denoted as

$$\mathcal{M}^p = \begin{pmatrix} \frac{1}{\delta t} Q_p & 0 & B_x^T \\ 0 & \frac{1}{\delta t} Q_p & B_y^T \\ B_x & B_y & 0 \end{pmatrix} \quad (4.8)$$

As we known, the schur complement of matrix  $\mathcal{M}^p$  is  $M_S = B_x Q_p^{-1} B_x^T + B_y Q_p^{-1} B_y^T$ . Referring to ([20], section 5), for LBB stable mixed approximations with enclosed flow boundary conditions,  $M_S$  is spectral equivalent with pressure Laplacian matrix  $A_p$ . So we use  $A_p$  to appropriate schur complement  $M_S$ . Then we choose the block triangular preconditioner

$$\mathcal{K} = \begin{pmatrix} Q_p & 0 & B_x^T \\ 0 & Q_p & B_y^T \\ 0 & 0 & M_S^* \end{pmatrix} \quad (4.9)$$

for (4.8), where  $M_S^* = A_p$  or  $M_S^* = \frac{1}{v} A_p$ . We will contrast the efficiency of the preconditioner by different choice of  $M_S^*$ . In our practical computation,  $\frac{1}{v} A_p$  performs more efficiently than  $A_p$ . For inflow/outflow problem, some modifications should be given for  $A_p$  on Neumann boundary to improve efficiency. Along the outflow boundary  $\partial \Omega_N$ , the discrete pressure  $p_h$  has to satisfy a homogeneous Dirichlet boundary condition. While a Neumann condition  $\frac{\partial p_h}{\partial n} = 0$  is needed along  $\partial \Omega_D$ , that means we do nothing on  $p_h$  along Dirichlet boundary. see [15] for detail. Note that all the matrixes  $M, B_x^T, B_y^T, B_x, B_y, A_p$  have to rebuilt once the meshes move.

In our algorithm, PCD preconditioned GMRES is selected as a solver solving linear system (3.2). We denote the stop criterion for GMRES convergence is

$$\|r^{(k)}\| \leq 10^{-6} \|r^{(0)}\| \quad (4.10)$$

where  $r^{(k)}$  is the residual of the linear system (3.2) and  $r^0$  is right hand side of (3.2). Finally, to illustrate our algorithm clearly, we give the flow-chart as the following algorithm 4.1:

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**Algorithm 4.1** Moving mesh FEM for Navier Stokes equation

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- 1: Solve steady Stokes flow to give the initial value  $\vec{u}_h^{(0)}, p_H^{(0)}$ .
  - 2: **while**  $t_n < T$  **do**
  - 3:     Calculate monitor function on mesh  $\Delta_p^{(n)}$  using  $\vec{u}_h^{(n)}, p_H^{(n)}$  and obtain logical mesh  $\xi^*$  by solving (4.2).
  - 4:     Judge if  $L_2$  norm of  $\xi^* - \xi^{(0)}$  is less than tolerance. If yes, the iterator is over, else continue 5 - 8.
  - 5:     Calculate move direction  $\delta \vec{x}$  of  $\Delta_p^{(n)}$  using the difference of  $\xi^* - \xi^{(0)}$ .
  - 6:     Solve equation (4.7) on  $\Delta_v^{(n)}$  to get medium variable  $\vec{u}_{h,*}^{(n)}, p_{H,*}^{(n)}$ .
  - 7:     Update mesh  $\Delta_p^{(n)}$  to  $\Delta_p^{(n+1)}$  and synchronize  $\Delta_v^{(n)}$  to  $\Delta_v^{(n+1)}$  by the hierarchy geometry tree structure.
  - 8:     Go back to 3.
  - 9:     Solve Navier-Stokes system (3.2) to obtain numerical solutions  $\vec{u}_h^{(n+1)}, p_H^{(n+1)}$  on mesh  $\Delta_v^{(n+1)}$  and  $\Delta_p^{(n+1)}$ .
  - 10: **end while**
- 

## 5. Numerical tests

We use three numerical tests to show our strategy. In practical computation, we choose the solutions of steady Stokes equations as the initial value of Navier-Stokes equations. The initial physical domain and logical domain in moving algorithm are the same. Moving mesh and numerical solutions are shown in below. Our codes are all based on the finite element package AFEPack.

### 5.1. Driven cavity flow

We consider the benchmark problem: regularized cavity flow. Our computational domain is  $\Omega = [-1, 1] \times [-1, 1]$  and viscosity is  $\nu = 0.001$ . Dirichlet boundary condition is imposed on  $\partial\Omega$ . At the top boundary,  $\vec{u} = (1 - x^4, 0)^T$  while no-slip boundary condition is setted on other parts of  $\partial\Omega$ .

In our moving strategy, (4.1) is selected as monitor function. Parameters  $\alpha = 0.5, \beta = 2.0$  perform well. The moving mesh and vorticity contour evolving to steady state are shown in Figure 3. It can be seen that mesh clusters at top boundary and right boundary



where the magnitude of vorticity is large. Velocity streamline is shown in Figure 5. We contrast the magnitude of velocity divergence between uniform mesh and moving mesh in Figure 4. It is found that the magnitude of velocity divergence in moving mesh is one half of uniform case. From Figure ??, it requires less GMRES iteration steps by choosing  $F_p = A_p + W_p^n$  than  $F_p = \nu A_p + W_p^n$  in PCD preconditioning. We compare the number of GMRES iteration counts in solving linear system (3.2) with AMG preconditioner and ILU preconditioner in Table 1. It is discovered that AMG is more efficient than ILU. We also compare the number of GMRES iteration steps in solving (4.7) with different preconditioning matrix  $M_S^*$ . It turns out to be that  $M_S^* = \frac{1}{\nu}$  is more efficient than  $M_S^* = A_p$  in solving (4.7).

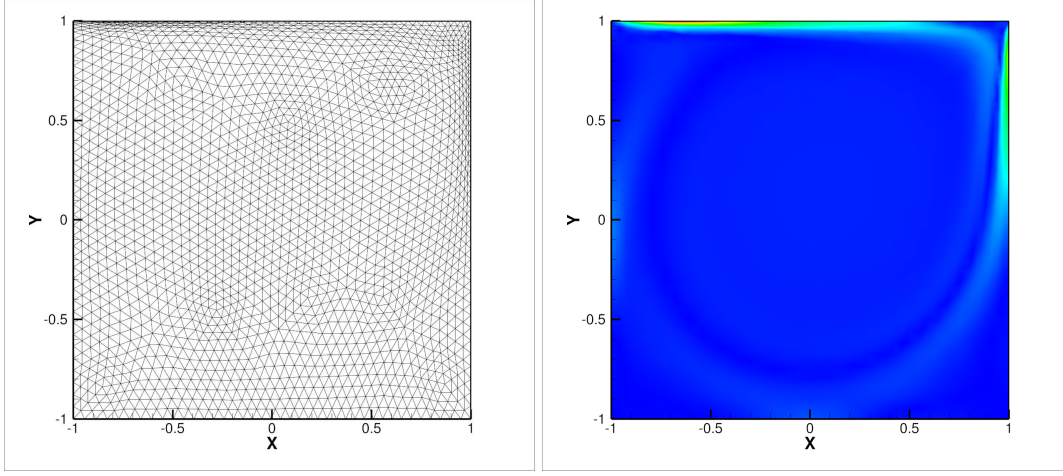


Figure 3: Cavity flow, left: mesh, right: vorticity contour, pressure mesh  $20 \times 20$ ,  $Re = 2000$ .

pressure mesh	mean GMRES step number	
	<i>AMG</i>	<i>ILU</i>
$20 \times 20$	10.8	107.5
$40 \times 40$	16.01	307.33
$80 \times 80$	27.18	> 500

Table 1: Cavity flow: comparison of mean GMRES step counts in solving linear system (3.2) with AMG and ILU preconditioning,  $Re = 2000$ .

## 5.2. Flow over cylinder

This example models the development of flow over an cylinder along a rectangular channel. This problem has been considered in [24]. The center of cylinder is  $(0, 0)$  and the radius is  $r = 0.3$ . Let viscosity  $\nu = 1/300$ , thus the Reynolds number  $Re = \frac{2rU_m}{\nu} = 240$

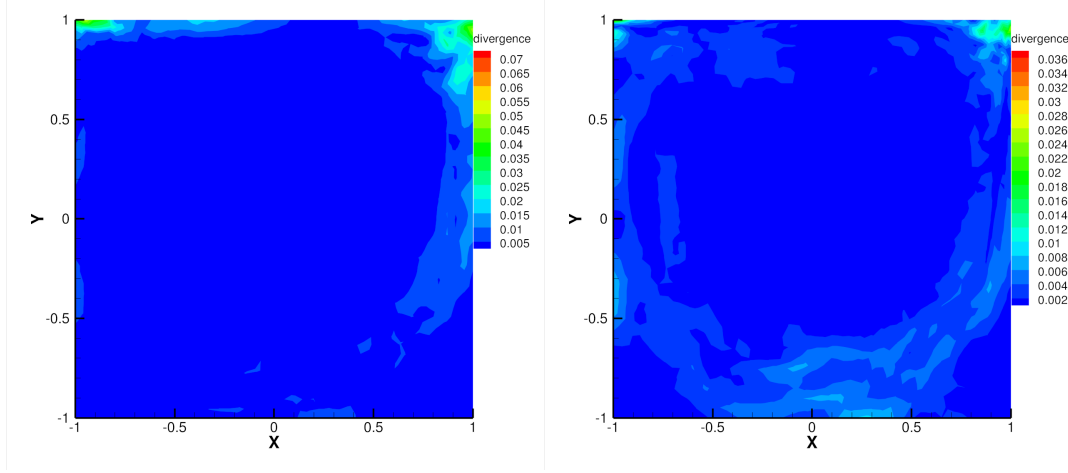


Figure 4: Cavity flow, divergence of velocity, left: uniform mesh, right: moving mesh, pressure mesh  $20 \times 20$ ,  $Re = 2000$ .

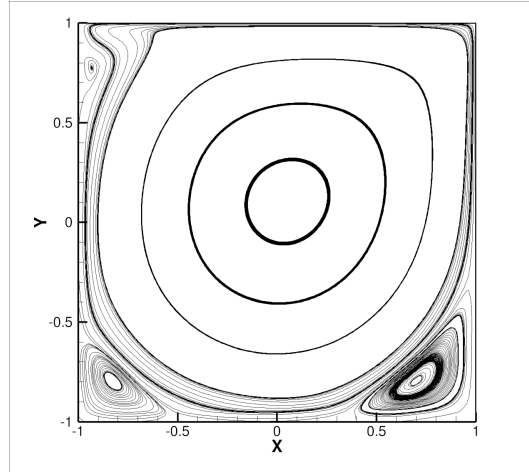


Figure 5: Cavity flow: velocity streamline, pressure mesh  $20 \times 20$ ,  $Re = 2000$ .

pressure mesh	GMRES step number	
	$M_S^* = \frac{1}{v}A_p$	$M_S^* = A_p$
$20 \times 20$	4.96	13.19
$40 \times 40$	10.99	42.36

Table 2: Cavity flow: the number of GMRES steps in solving (4.7) in solution redistribution with different precondition matrix  $M_S^*$ ,  $Re = 2000$ .

by direct computation, where  $U_m$  is the mean velocity of inflow. The domain is  $\Omega = [-1, 5] \times [-1, 1]$ . At the inflow boundary  $x = -1$ ,  $\vec{u} = (1 - y^2, 0)^T$  with poiseuille profile is imposed. On the top and bottom boundary of the channel, condition  $\vec{u} = (0, 0)^T$  is settled. Natural condition is imposed on  $x = 5$ .

In our moving strategy, parameters  $\alpha$  and  $\beta$  in (4.1) are user defined. The value of  $\alpha$  is greater, the degree of mesh clustering is larger. From Figure 6, it can be shown that the number of GMRES iteration step with  $\alpha = 5$  is larger than  $\alpha = 1.0$ . Comparison of GMRES step counts between different choices of  $F_p$  are shown in Figure 6, it is found that the number of GMRES iteration steps will decrease more than 20 by using  $F_p = A + W_p^n$ .

We show the moving mesh at  $t = 2s$  in Figure 7. It can be seen that the mesh obviously clusters around the cylinder. As we known, wall street phenomena will occur as time evolving when the flow has an appropriate viscosity according to [25], just as the mesh shown in Figure 8.

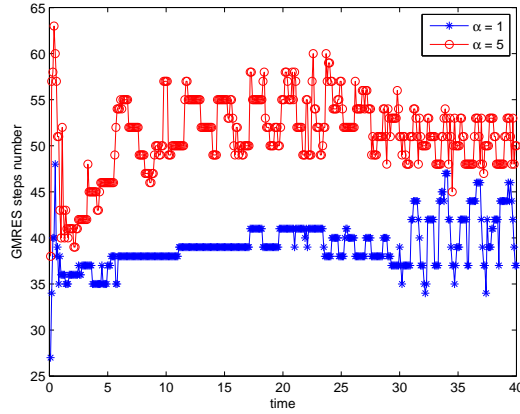


Figure 6: Flow over cylinder: GMRES iteration counts of solving(3.2) using PCD preconditioning with different  $\alpha$ ,  $Re = 240$ .

linear system	number of GMRES steps			
	$F_p = A + W_p^n$		$F_p = \nu A + W_p^n$	
	$M_S^* = \frac{1}{\nu}A_p$	$M_S^* = A_p$	$M_S^* = \frac{1}{\nu}A_p$	$M_S^* = A_p$
(3.2)	51.06	45.13	0	61.08
(4.7)	22.25	40.35	0	40.78

Table 3: Flow over cylinder: the number of GMRES steps in solving (3.2) and (4.7) with different precondition matrix  $M_S^*$  and different  $F_p$  in PCD preconditioning,  $\alpha = 5, Re = 240$ .

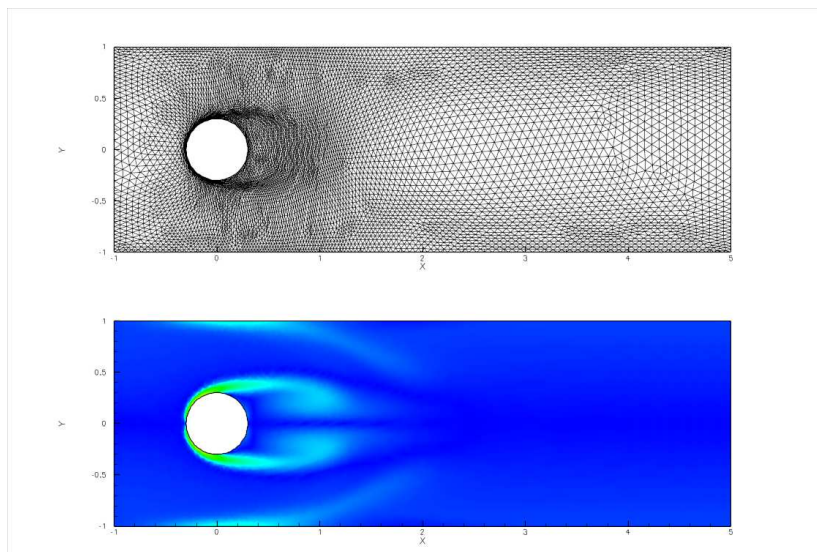


Figure 7: Flow over cylinder: moving mesh at  $t = 2s$ ,  $Re = 240$ .

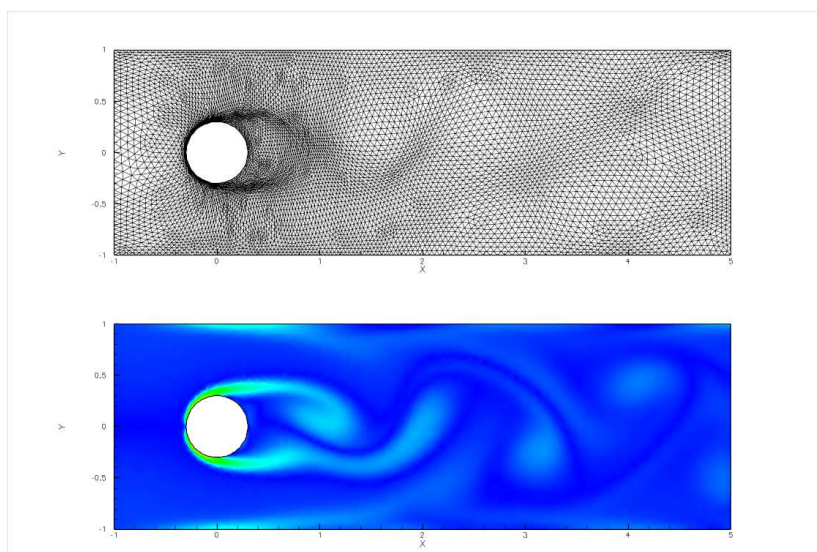


Figure 8: Flow over cylinder: moving mesh at  $t = 40s$ ,  $Re = 240$ .

## 6. Remarks

In this work, we apply an efficient AMG preconditioning strategy to moving mesh finite element method based on  $4P1 - P1$  pair. The  $4P1 - P1$  element pair naturally satisfies the inf-sup condition and is linear-order. Linear element is more preferred than high order element in practical engineering computation, according to its simplicity and complexities of problems. In our moving strategy, we use the monitor function based on vorticity to capture the fine flow structure. The structure of mesh is consistent with vorticity structure. We compare the number of GMRES iteration steps in solving Navier-Stokes problems with different  $F_p$  in PCD preconditioning. It is verified that choosing  $F_p = A_p + W_p^n$  in PCD preconditioning is more efficient by three numerical tests. We also contrast the number of GMRES step counts of solving linear system in solution re-distribution with different preconditioning matrix  $M_S^*$ . It turns out to be that  $M_S^* = \frac{1}{v}A_p$  is more efficient. We find that the number of GMRES iteration steps will be larger as the mesh becomes clustering.

We will extend the efficient preconditioning to some interested problems such as free boundary problem. Also three dimension problems of solving incompressible flow with moving mesh finite element based on  $4P1 - P1$  pair will be considered in future work.

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