University of California, Berkeley Department of Mechanical Engineering ME 132 Dynamic Systems and Feedback

Fall 2013 Midterm II

Closed book and closed lecture notes;
Two 8.5×11 handwritten summary sheets allowed;
Scientific calculator with no graphics allowed;
Keep your phones in your bag, and set them off or in silent mod

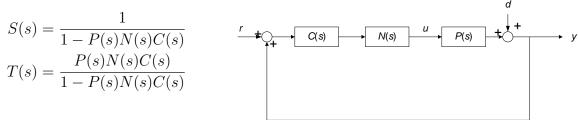
NAME:	
STUDENT ID #:	

#1	#2	#3	#4	Total
10	10	10	20	50

Instructions:

- Write down your name and student ID.
- Read the problems carefully.
- Write your solutions clearly.
- Turn in your exam ON TIME. If we don't have your exam to grade, we will have to put a zero on the score sheet.
- This exam has 9 pages. The last page is empty.
- Do not turn to the next page until you are instructed to do so.

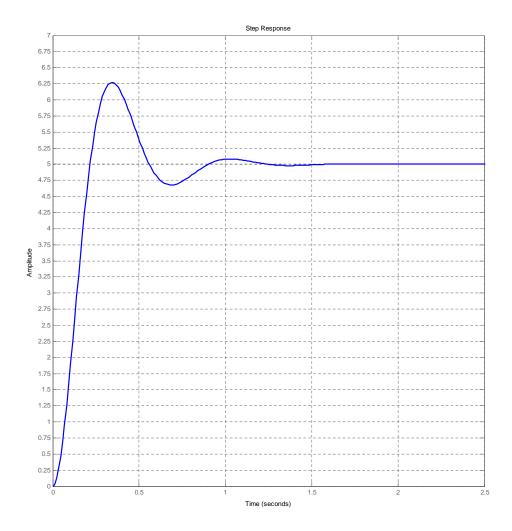
- 1. [10 points] True or false. Circle your choice in the provided table below. For the false statements, provide a one-sentence reasoning (remark: correct judgments with incorrect or no reasoning get zero point).
 - (a) A standard PID controller is causal
 - (b) In PID design, the integral action removes the steady-state error for step inputs
 - (c) Modern control theory is based on state-space analysis
 - (d) In PID design, the integral action perfectly rejects constant disturbances at the steady state
 - (e) In PID design, high-gain feedback at low frequency comes from the "D term"
 - (f) With a well-designed feedback controller, we can achieve good rejection for both the disturbance and the sensor noise at a particular frequency
 - (g) When modeling a moving mass using state-space equations, it is reasonable to select the position and the negative of the position as the state variables
 - (h) First-order LTI systems can have resonant behavior
 - (i) The sensitivity function directly explains the disturbance rejection property of a feedback system
 - (j) For the block diagram below, the sensitivity and complementary sensitivity functions are respectively



Circle your choice and provide the reasoning for the false statements:

- (a): true, false
- (b): true, false
- (c): true, false
- (d): true, false
- (e): true, false
- (f): true, false
- (g): true, false
- (h): true, false
- (i): true, false
- (j): true, false

- 2. [10 points] System analysis.
 - (a) [4 points] Shown below is a time-domain step response of a linear time invariant system.



- i. Mark the 5% settling time and peak time on the figure. Compute the maximum overshoot M_p . $M_p =$
- ii. From the following four transfer functions,

$$\frac{500}{s^2 + 8s + 100}, \ \frac{500}{s^2 + 0.8s + 100}, \ \frac{100 - 400s}{s^2 + 8s + 100}, \ \frac{100000}{\left(s^2 + 8s + 100\right)\left(s + 200\right)}$$

circle the one(s) that may have generated the step response. Provide your reasoning.

3

(b) [6 points] A set of unit step responses $\{G[1,1]\ G[1,2]\ G[2,1]\ G[2,2]\ G[3,1]\ G[3,2]\}$ and six transfer functions $(A\ to\ F)$ are provided below. In the provided box, match each step response with its corresponding transfer function. Assume zero initial condition in all systems from A to F.

 $A: \ \tfrac{1}{s+1}$

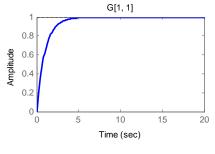
 $B: \frac{-s}{s^2 + 2 \times 0.5s + 1}$

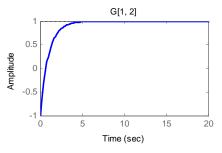
 $C: \ \frac{1}{s^2+1}$

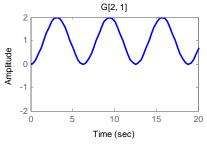
 $D: \frac{s+1}{s^2+2\times 0.5s+1}$

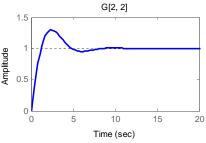
 $E: \frac{1-s}{s+1}$

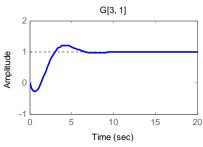
 $F: \frac{-s+1}{s^2+2\times 0.5s+1}$

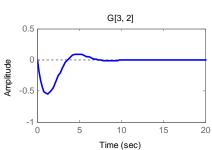












Answer (fill in the blanks):

A matches $G[\ __,__\];\, B$ matches $G[\ __,__\]$

C matches $G[\ __,__\];\, D$ matches $G[\ __,__\]$

E matches $G[__,__]$; F matches $G[__,__]$

a)	of you solutions in (a)-(c) should contain two free constants. [2 points] $\ddot{y}(t) + 5\dot{y}(t) + 6y(t) = 0$
	Final result:
b)	[2 points] $\ddot{y}(t) + 5\dot{y}(t) + 6y(t) = 18u(t)$ where $u(t)$ is a standard unit step signal.
	Final result:
(c)	[2 points] $\ddot{y}(t) + 5\dot{y}(t) + 6y(t) = 18u(t) - 18\dot{u}(t)$ where $u(t)$ is a standard unit step sign
	Final result:
	[4 points] Obtain the full solution of (c) if the initial conditions are $y(0) = 0$ and $\dot{y}(0)$. The result should contain no undetermined coefficients.

- 4. [20 points] System design and analysis.
 - (a) [5 points] Consider the state-space description of a system

$$\dot{x} = \begin{bmatrix} 0 & 1 \\ -100 & -4 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} r$$

$$y = \begin{bmatrix} 100 & 3 \end{bmatrix} x + 2r$$

i. Obtain the transfer function from r to y.

Final result:

$$G_{r \to y}(s) =$$

ii. Now consider another system

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ -100 & -4 & 1 \\ \hline 100 & 3 & 2 \\ 0 & 100 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \hline r \end{bmatrix}$$

Obtain the transfer function from r to y_1 .

Final result:

$$G_{r \to y_1}(s) =$$

(b) [2 points] Suppose the derived transfer function in part (a)-i is

$$G_{r \to y}(s) = \frac{24s + 100}{s^2 + 25s + 100} \tag{1}$$

(warning: this is not the right answer for part (a)-i. Do NOT copy it to the answer box there). Equation (1) is the closed-loop transfer function from r to y in the feedback system in Fig. 1. Derive k_P and k_I , the coefficients of the PI controller.

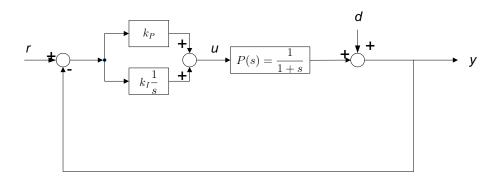


Figure 1: Block diagram for problem 4

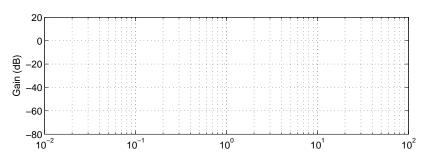
Final result: $k_P =$ $k_I =$

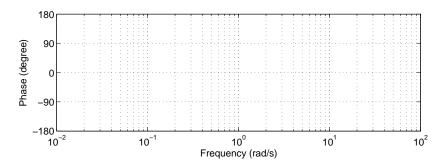
(c) [5 points] Sketch, on the provided figure, the magnitude responses of

$$P(s) = \frac{1}{1+s}, \quad G_{r\to y}(s) = \frac{24s+100}{s^2+25s+100}$$
 (2)

and the phase response of

$$P\left(s\right) = \frac{1}{1+s}$$





(d) [2 points] Consider the step responses of the two systems $G_{r\to y}(s)$ and P(s) in Equation (2). Assume zero initial condition(s). Which one has shorter transient response? Provide your reasoning.

Conclusion (circle one):

 $G_{r\to y}(s)$ has shorter transient

P(s) has shorter transient

(e) [6 points] Consider the sinusoidal responses of the two systems $G_{r\to y}(s)$ and P(s) in Equation (2), with respect to the input $u(t) = \sin\left(3t + \frac{\pi}{4}\right)$, $t \ge 0$ (u(t) = 0 if t < 0). At steady state, which system provides better tracking to u(t)? Derive this steady-state output.

Remark: you don't have to compute the numerical values of the magnitude and phase.

Conclusion (circle one):

 $G_{r\to y}(s)$ is better

P(s) is better