

NUCLEAR ENGINEERING, UC BERKELEY

---

# ME 280A Finite Element Analysis HOMEWORK 4: ERROR ESTIMATION and ADAPTIVE MESHING

---

Xin Wang

October 25, 2014

# 1 INTRODUCTION

The objective of this project is to solve two 1-D differential equation systems using linear elements.

$$\begin{aligned}\frac{d}{dx}(A_1(x)\frac{du}{dx}) &= f(x) \\ f(x) &= 90\pi^2 \sin(3\pi x) \sin(36\pi x^3) - (10\sin(3\pi x) + 5)... \\ (216\pi x \cos(36\pi x^3) - 11664\pi^2 x^4 \sin(36\pi x^3)) - 6480\pi^2 x^2 \cos(3\pi x) \cos(36\pi x^3) \\ A_1(x) &= 1.0 \\ L &= 1 \\ u(0) &= 0 \\ u(L) &= 0 \\ u^{true} &= (10\sin(3\pi x) + 5)\sin(36\pi x^3) \\ du/dx^{true} &= 6\pi(5\sin(36\pi x^3)\cos(3\pi x) + 90x^2(2\sin(3\pi x) + 1)\cos(36\pi x^3))\end{aligned}\tag{1}$$

and

$$\begin{aligned}
\frac{d}{dx}(A_1(x)\frac{du}{dx}) &= f(x) \\
f(x) &= 256\sin(\frac{3\pi x}{4})\cos(16\pi x) \\
A_1(x) &= \begin{cases} 0.2 & x < 1/3 \\ 2.0 & x \geq 1/3 \end{cases} \\
L &= 1 \\
u(0) &= 0 \\
A_1(L)\frac{du}{dx}(L) &= 1 \\
u^{true} &= \frac{512}{4087\pi A_1} \left( \frac{268\sin(\frac{61\pi x}{4})}{61\pi} - \frac{244\sin(\frac{67\pi x}{4})}{67\pi} \right) \\
&+ \begin{cases} (5 + \frac{7680\sqrt{2}}{4087\pi})x, & x < 1/3 \\ \frac{2304\sqrt{2}(8210-768\sqrt{3}+4087\pi)}{16703569\pi^2} + \frac{3}{2} + (\frac{1}{2} + \frac{768\sqrt{2}}{4087\pi})x, & x \geq 1/3 \end{cases} \\
du/dx^{true} &= \frac{512}{4087\pi A_1} (67\cos(\frac{61\pi x}{4}) - 61\cos(\frac{67\pi x}{4})) \\
&+ \begin{cases} (5 + \frac{7680\sqrt{2}}{4087\pi}), & x < 1/3 \\ (\frac{1}{2} + \frac{768\sqrt{2}}{4087\pi}), & x \geq 1/3 \end{cases} \tag{2}
\end{aligned}$$

## 2 FINITE ELEMENT METHOD

### 2.1 WEAK FORM

## 3 ADAPTIVE MESHING

### 3.1 DATA STRUCTURE

The node coordinates are stored in a  $N \times 1$  table. And the connectivity table contains the node numbers for each element, i.e  $ce = connectivity(e) = [ce(1), ce(2)]$  for linear element, where  $ce(1)$  and  $ce(2)$  are the number of the nodes at the element  $e$ .

The stiffness matrix  $K$  is no more a band matrix, but instead filled as:

$$\begin{aligned} K[ce(i), ce(j)] &= Ke(i, j) \\ i &= 1, 2; j = 1, 2 \end{aligned} \quad (3)$$

The load matrix is not filled in order anymore but still a  $1 \times N$  array:

$$\begin{aligned} R(ce(i)) &= Re(i) \\ i &= 1, 2 \end{aligned} \quad (4)$$

### 3.2 ADAPTIVE MESHING PROCESS

The process to solve the problem using adaptive meshing is:

- solve with an initial mesh,  $N_e = 16$
- loop through the mesh and calculate the element error
  - fetch the node numbers in this element using connectivity table  
ce= [2, 5]
  - fetch xe and ae with ce
  - compute  $E_e$  with gaussian quadrature
- if the error is larger than the tolerance for an element, i.e.  $E_e > tol_e$ , then divide the element into 2
  - add the new node  $nodex(end+1) = (xe(1)+xe(2))/2$
  - $connectivity(e) = [ce(1), len(nodex)]$
  - add a new element:  $connectivity(end+1) = [len(nodex), ce(2)]$
  - changed element = true
- check if any element was modified, if so, resolve with the new mesh, otherwise done.

The error is calculated on every mesh element as:

$$\begin{aligned}
E_I &= \frac{\frac{1}{h_I} \|u - u^N\|_{A_1(\Omega_I)}^2}{\frac{1}{L} \|u\|_{A_1(\Omega)}^2} \\
&= \frac{\frac{1}{h_I} \int_{\Omega_I} A_1(x) \left( \frac{du^{True}}{dx} - \frac{du}{dx} \right)^2 dx}{\frac{1}{L} \int_{\Omega} A_1(x) \left( \frac{du^{True}}{dx} \right)^2 dx}
\end{aligned} \tag{5}$$

Etol is defined as the overall tolerance divided by the number of elements.

## 4 ERROR CALCULATIONS

The error is defined as

$$\begin{aligned}
e^N &= \frac{\|u - u^N\|_{A_1(\Omega)}}{\|u\|_{A_1(\Omega)}} \\
\|u\|_{A_1(\Omega)} &= \sqrt{\int_{\Omega} \frac{du}{dx} A_1 \frac{du}{dx} dx} = \sqrt{\int_{\Omega} A_1 \left( \frac{du}{dx} \right)^2 dx} \\
\|u - u^N\|_{A_1(\Omega)} &= \sqrt{\int_{\Omega} A_1 \left( \frac{d(u - u^N)}{dx} \right)^2 dx}
\end{aligned} \tag{6}$$

To compute the error numerically, we calculate the two following quantities element by element, and then assemble them to obtain the overall error:

$$\begin{aligned}
\|u\|_{A_1(x)(\Omega)}^2 &= \int_{\Omega} A_1(x) \left( \frac{du}{dx} \right)^2 dx \\
\|u - u^N\|_{A_1(x)(\Omega)}^2 &= \int_{\Omega} A_1(x) \left( \frac{d(u - u^N)}{dx} \right)^2 dx \\
&= \int_{\Omega} A_1(x) \left( \frac{du}{dx} - \frac{du_N}{dx} \right)^2 dx \\
&= \sum_{e=1}^{N_e} \int_{x_e}^{x_{e+1}} A_1(x) \left( \frac{du}{dx} - \frac{du_N}{dx} \right)^2 dx \\
&= \sum_{e=1}^{N_e} \int_{-1}^1 A_1(x(\xi)) \left( \frac{du}{d\xi} - \frac{du_N}{d\xi} \right)^2 J d\xi
\end{aligned} \tag{7}$$

## 5 RESULTS

### 5.1 EQUALLY SPACED LINEAR ELEMENTS

For each problem, the minimum number of equally spaced linear elements needed in order to achieve  $e^N \leq 0.05$  are 1466 and 382.

For the second problem, the left side of the domains was divided into 85 elements and the right side into 171 elements.

### 5.2 ADAPTIVE MESHING

Solving both adaptive meshing was also used to solve the equation.

## 6 CONCLUSION