

# 第四章 导热问题的数值解

——复杂导热问题的近似解法

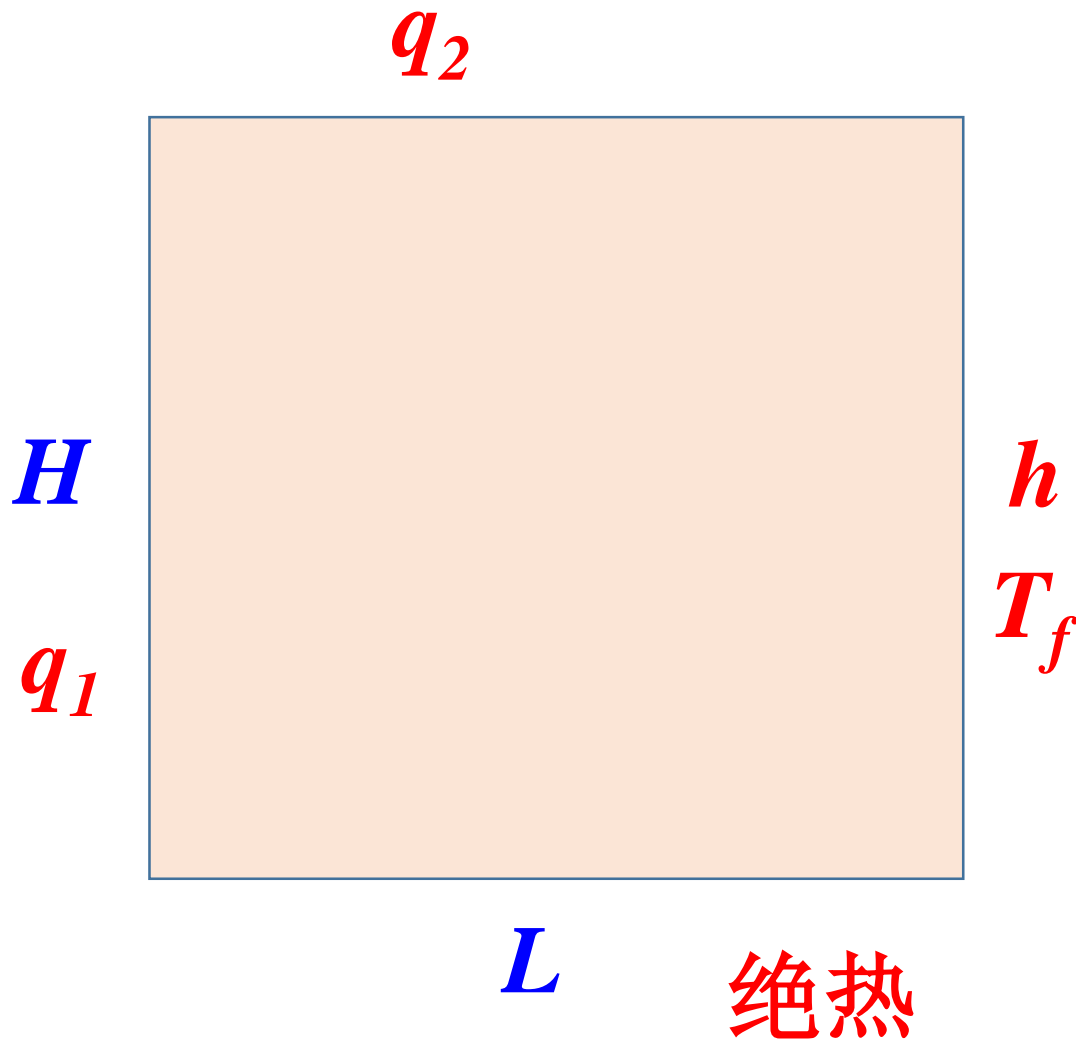
# 非稳态热传导

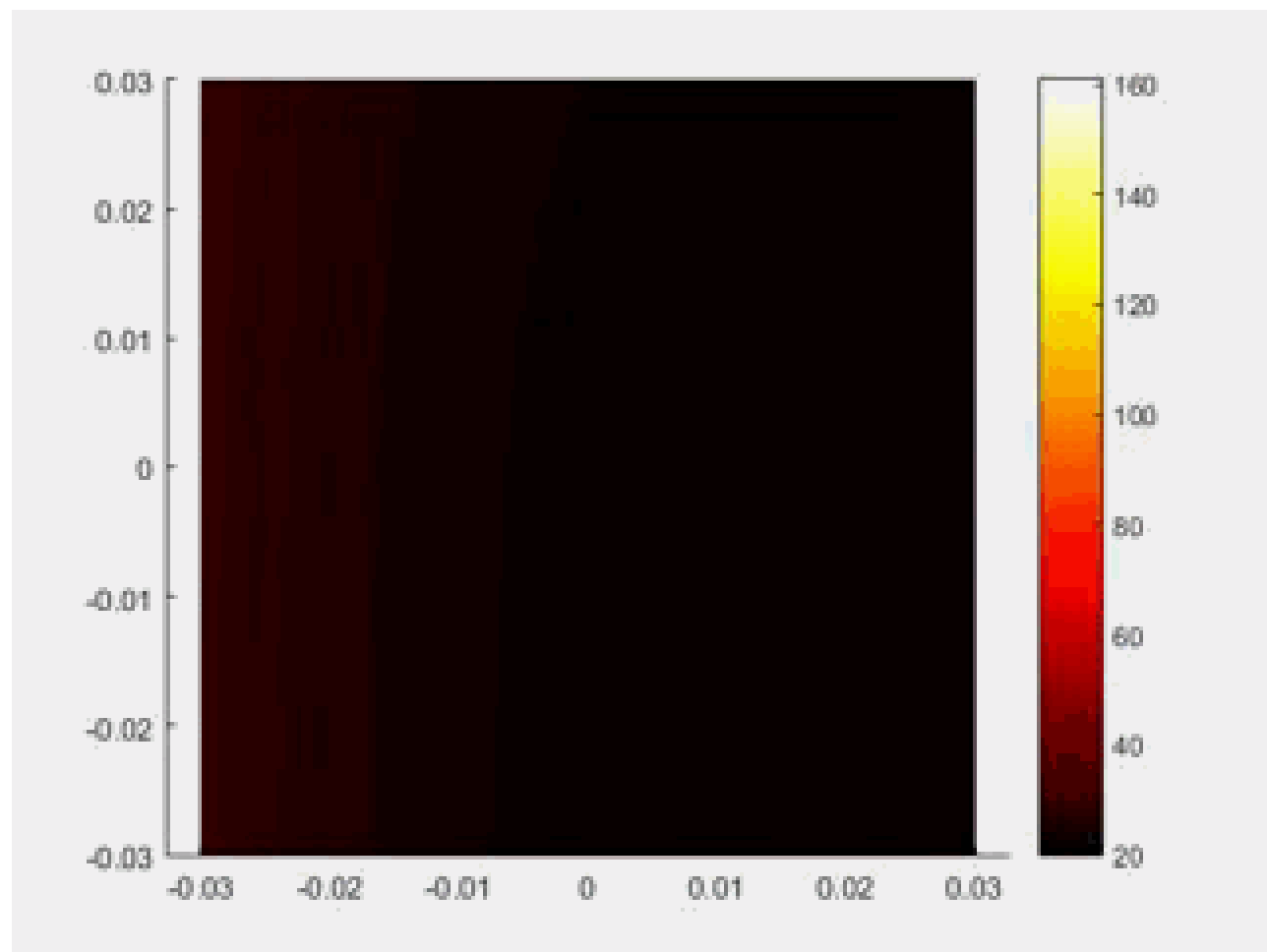
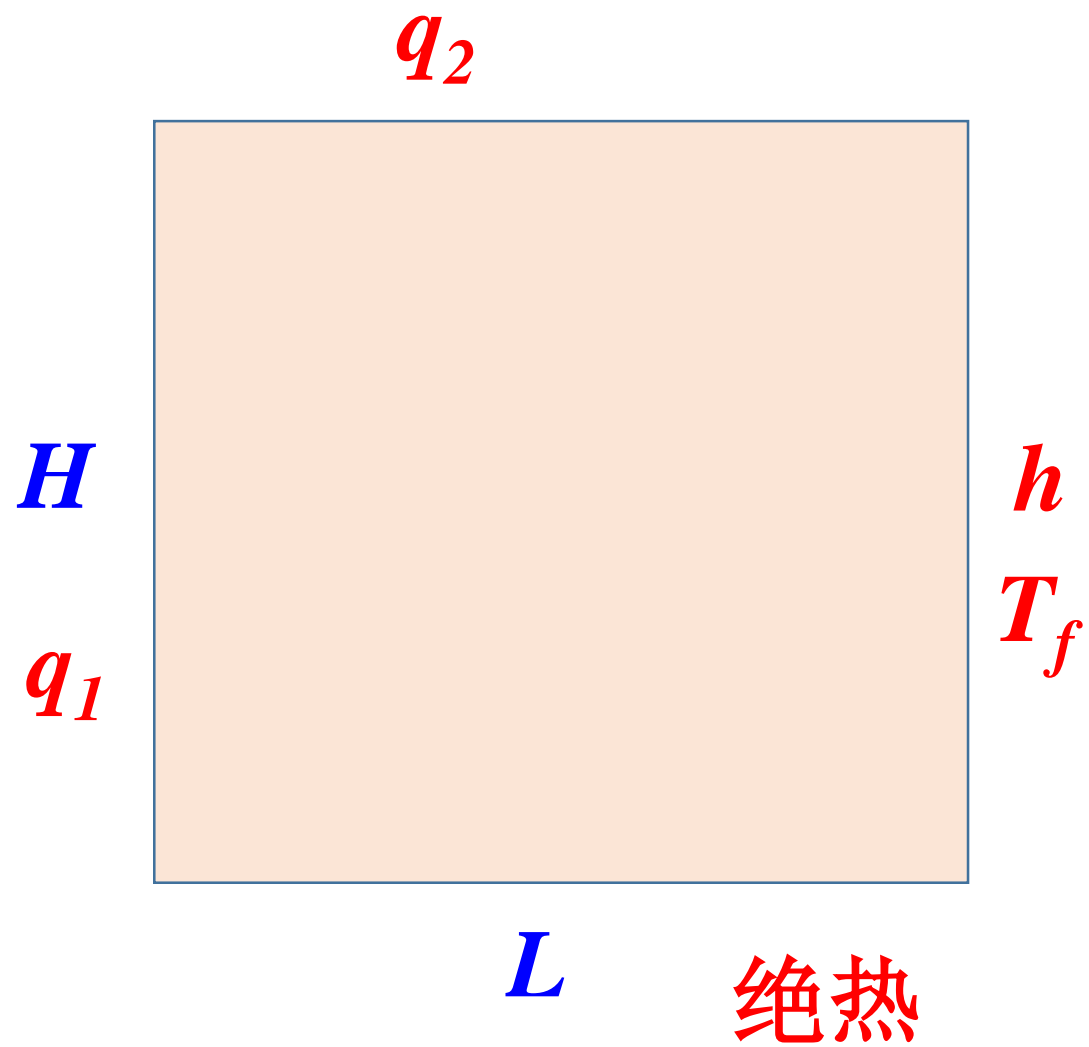
- 无限大平板的非稳态导热
- 集总参数法
- 半无限大物体的非稳态导热
- 第三类边界条件下的二维和三维非稳态导热

# 实际计算案例

一个金属固体，长6cm，宽6cm，垂直纸面方向无限长，左端面边界条件：

$q_1=10 \text{ W/cm}^2$ ，下端面绝热，上端面边界条件： $q_2=1 \text{ W/cm}^2$ ，右端面边界条件： $h=1000 \text{ W/m}^2 \text{ K}$ ， $T_f=20^\circ\text{C}$ ，导热系数 $\lambda=200 \text{ W/m K}$ ，





热传导  
求解方法



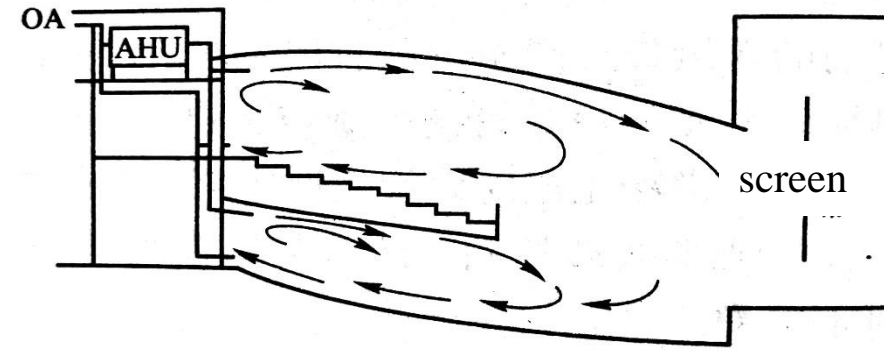
理论分析

实验方法

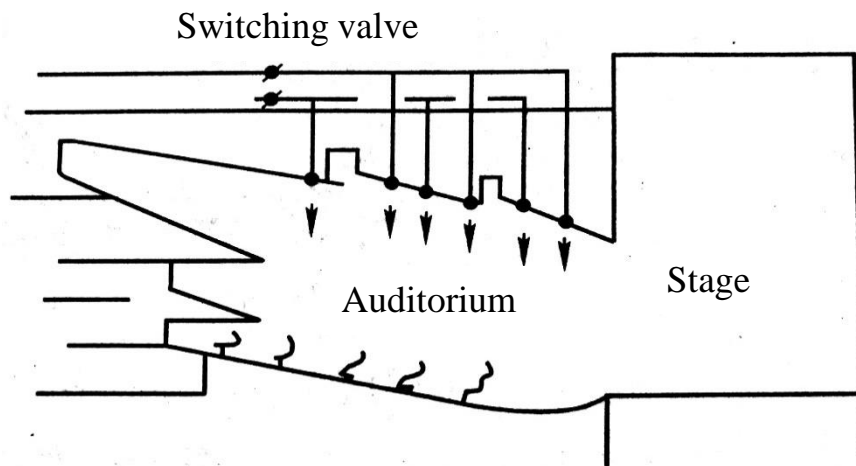
数值计算

- 几何条件复杂
- 边界条件复杂
-

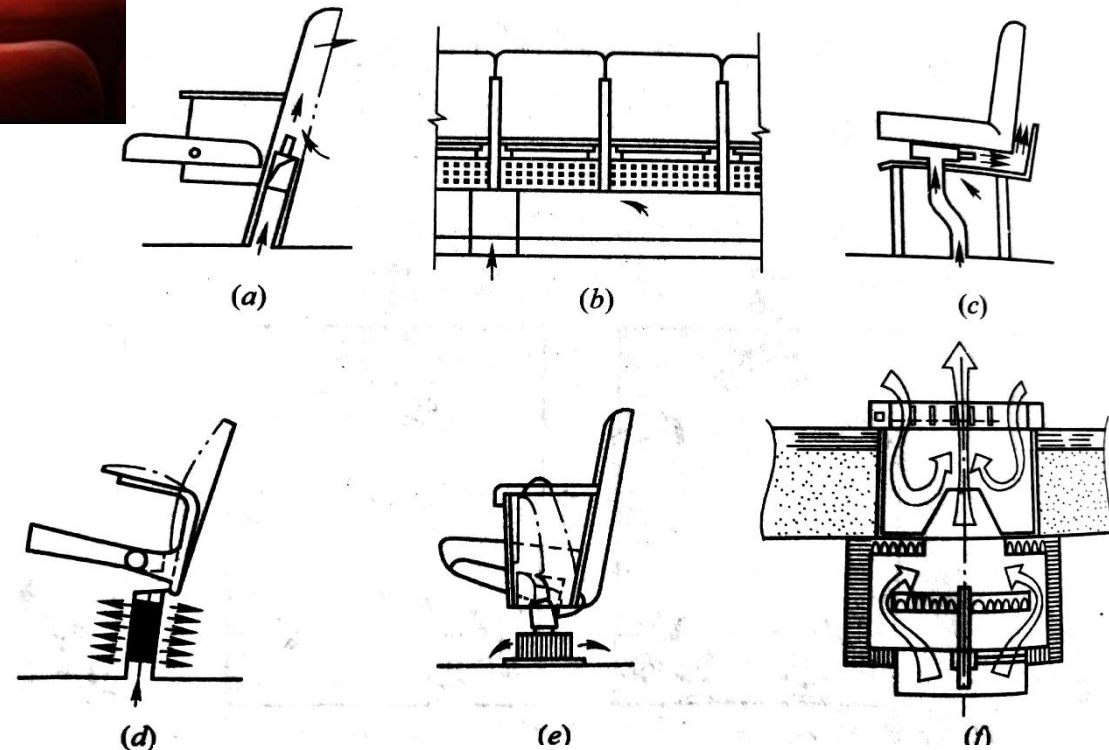
# Radiation cooling technology in Theater (剧场辐射制冷技术)



Nozzle deliver model

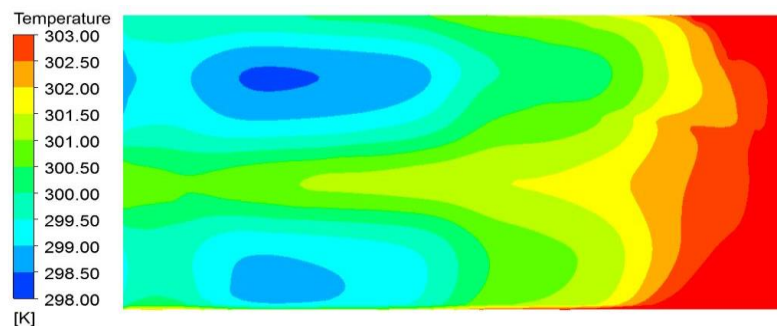


Top-supply and down-return air supply model

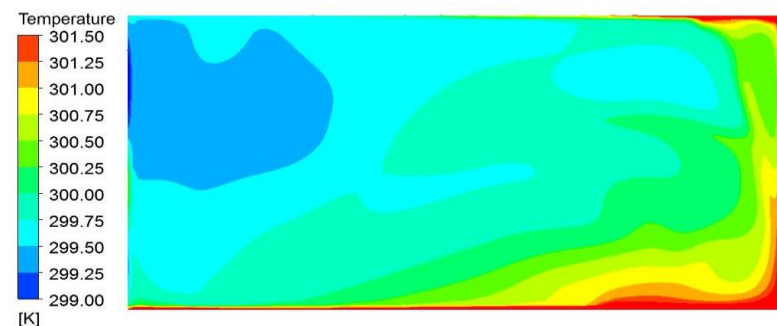


## ■ Distribution of indoor temperature

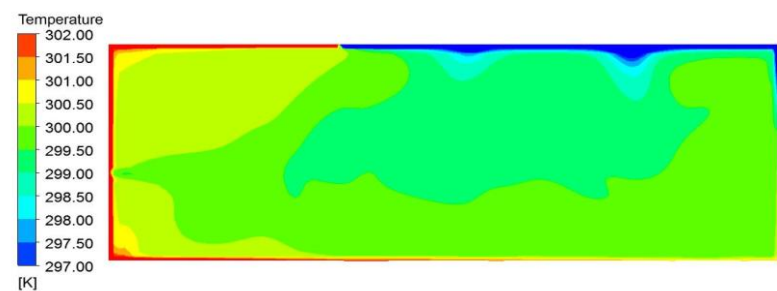
### ➤ Roof radiation system (顶部)



Y=0.1m

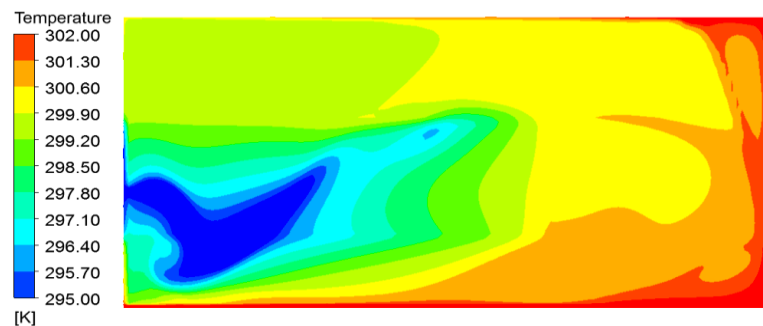


Y=1.1m

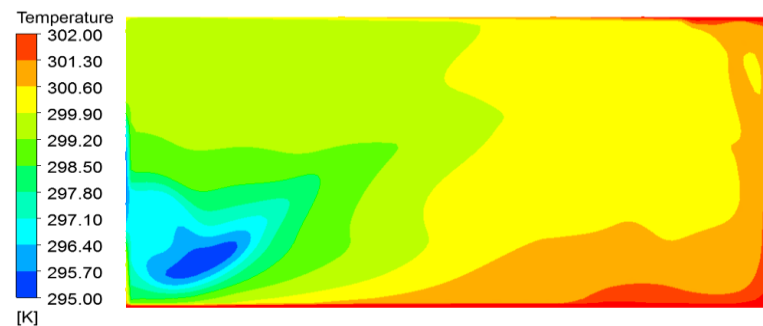


Z=10m

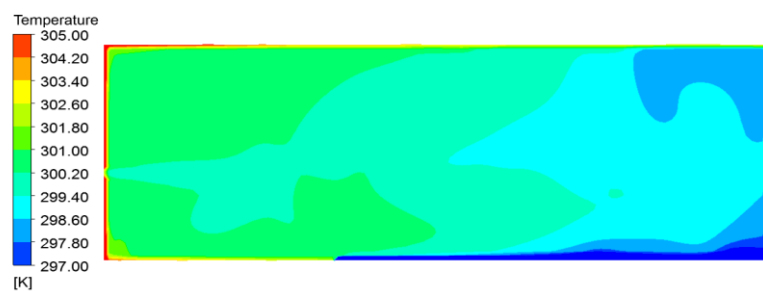
### ➤ Floor radiation system (地面)



Y=0.1m

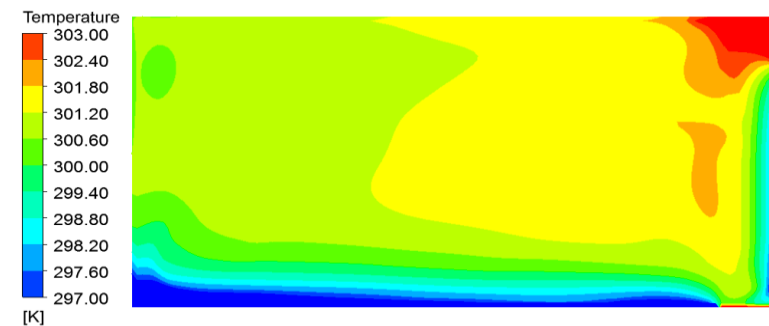


Y=1.1m

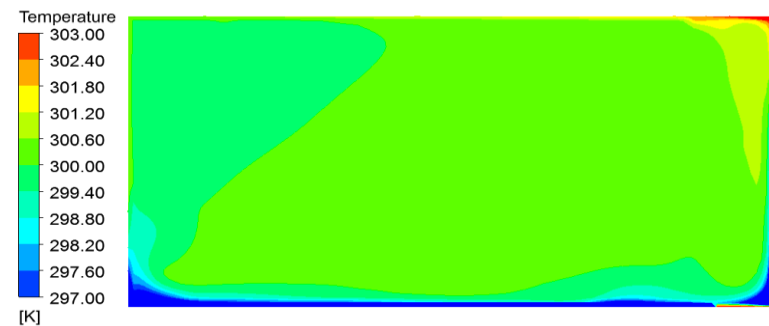


Z=10m

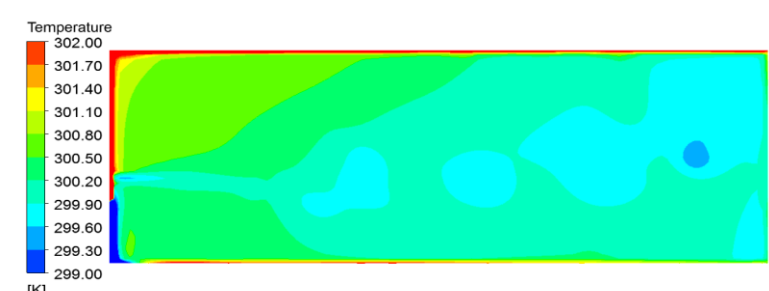
### ➤ Wall radiation system (壁面)



Y=0.1m



Y=1.1m



Z=10m

## 热传导 求解方法



理论分析

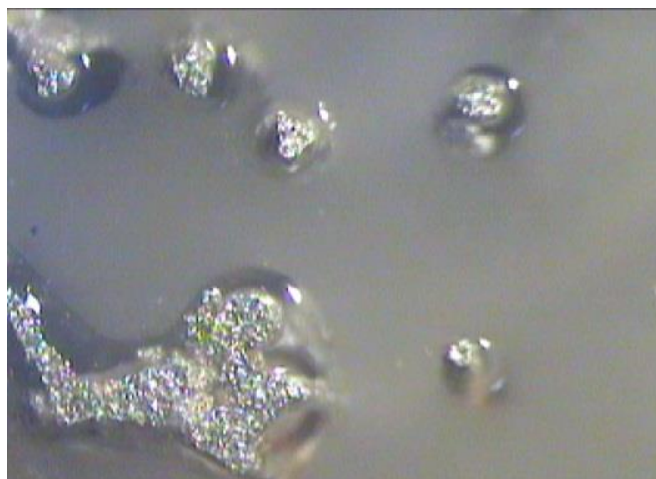
实验方法

数值计算

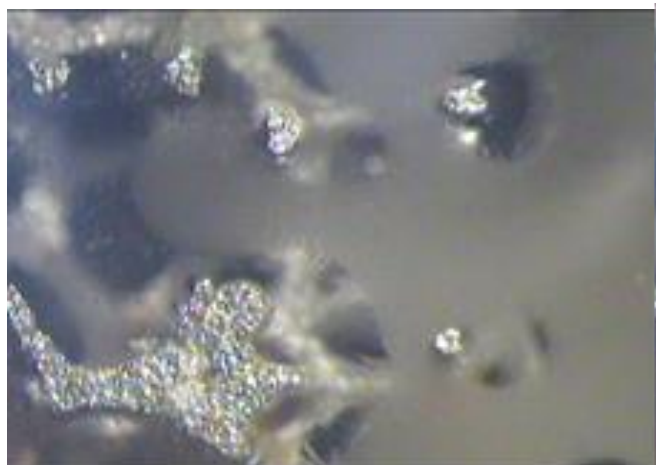
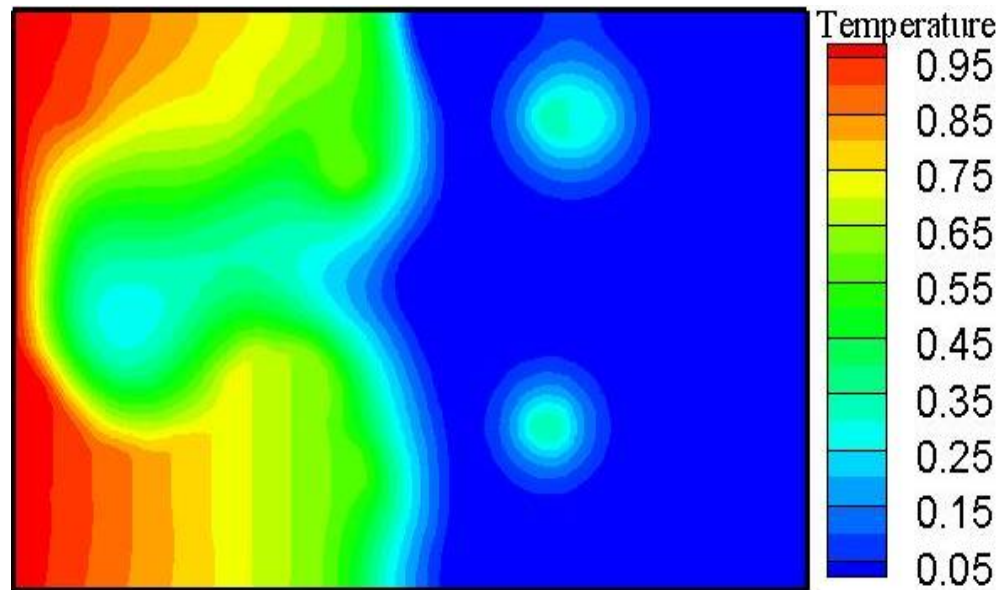
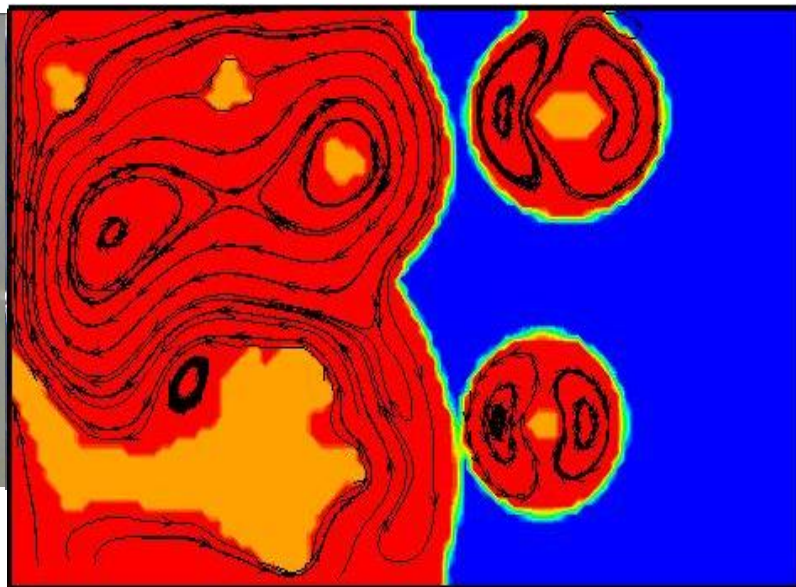
- 几何条件复杂
- 边界条件复杂
- 热物性随位置变化
- 热物性随温度变化
- 移动边界



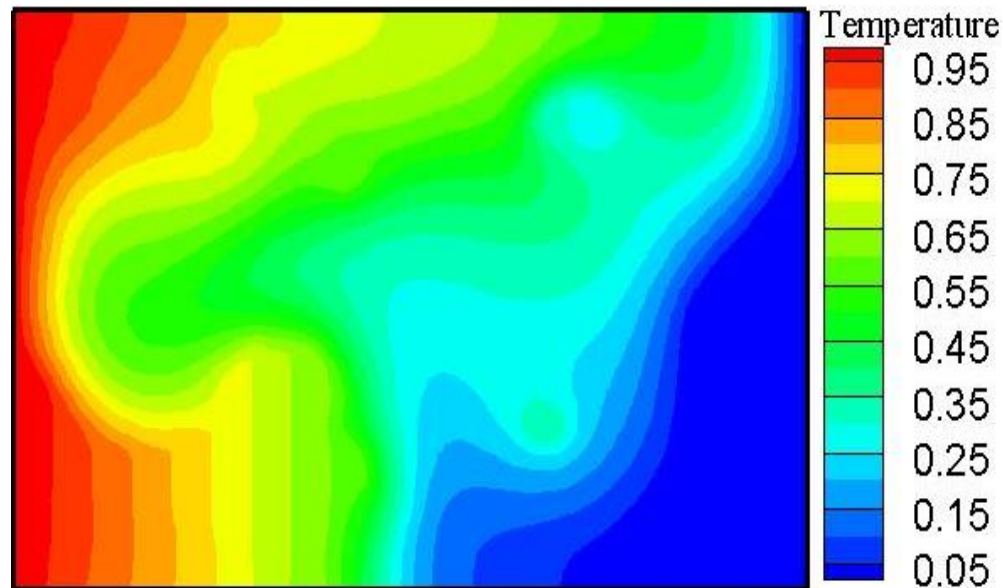
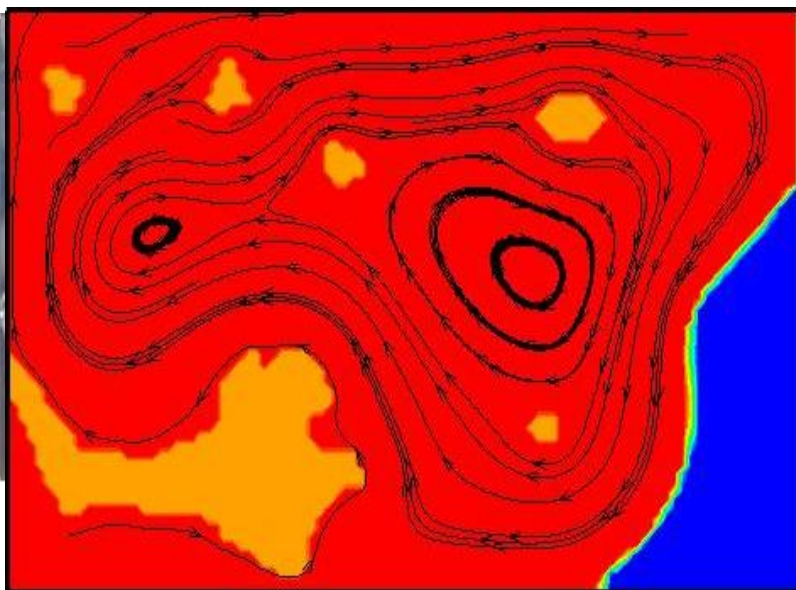
# 泡沫金属内石蜡融化过程



0 min

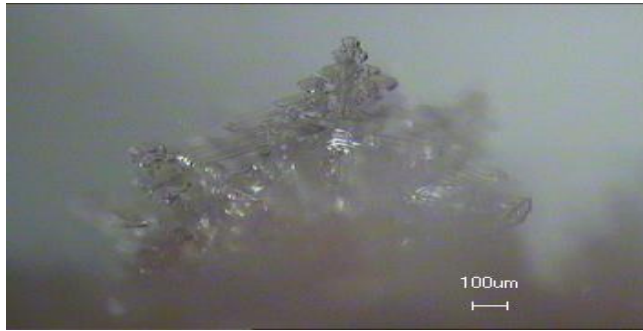
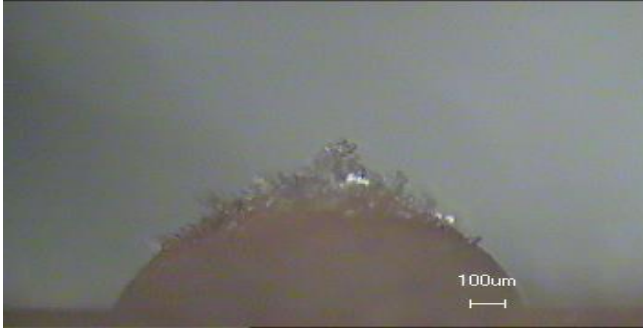
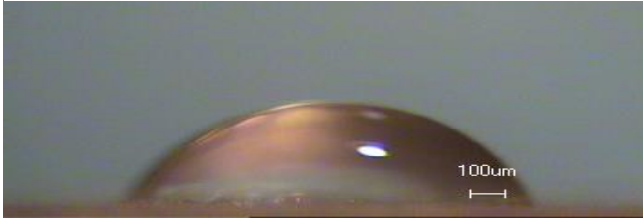


27 min

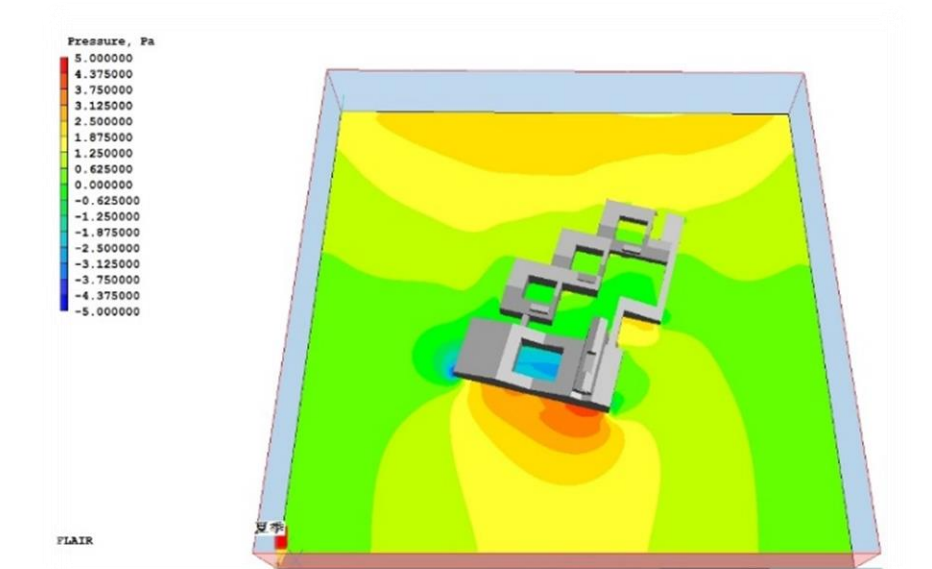
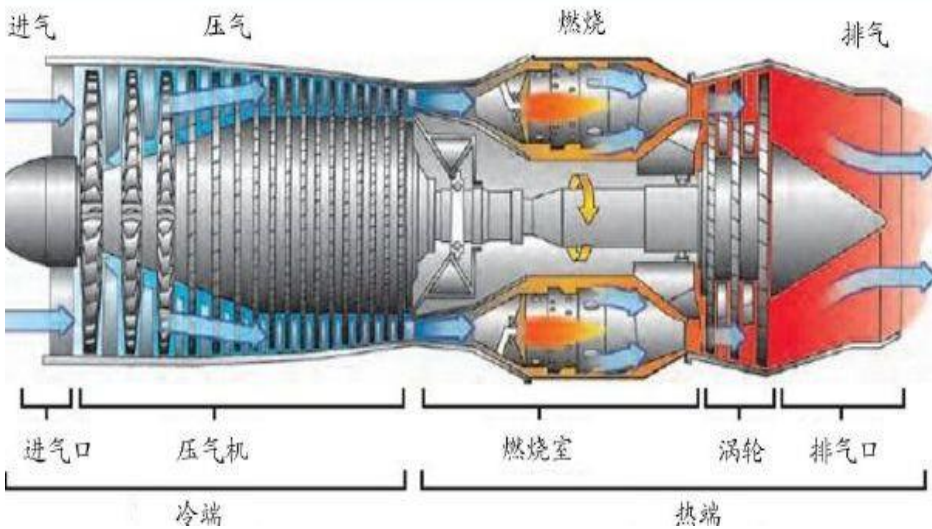
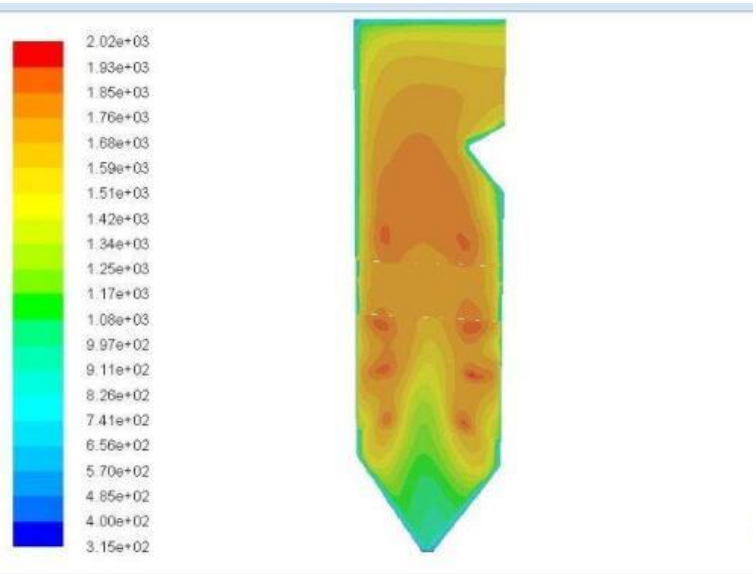




# 冰晶生成过程观测

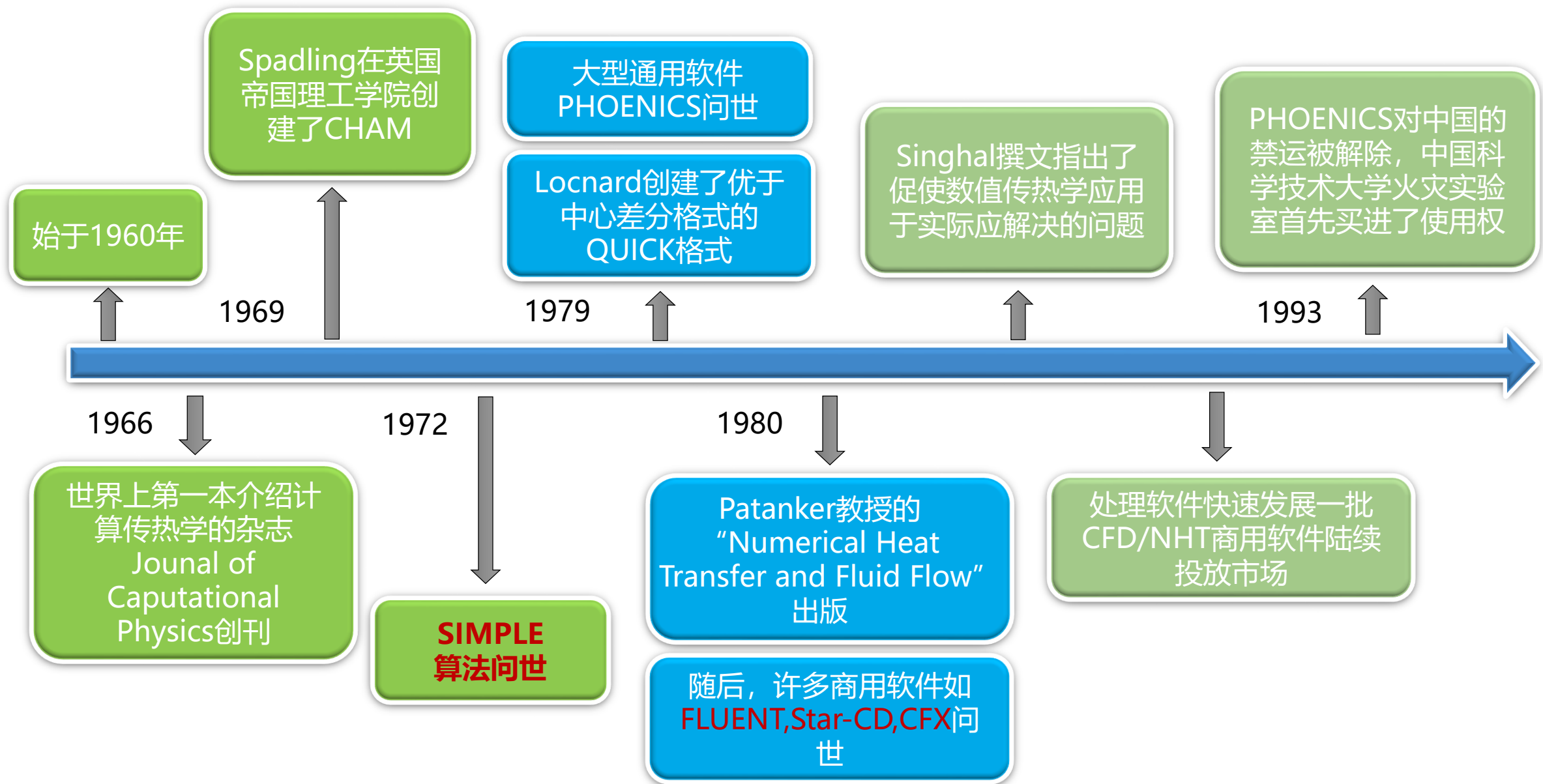


# 计算传热学的应用





# 计算传热学发展简史



## 4.1 数值求解的基本思想

## 4.2 离散方程的建立方法

## 4.3 二维稳态导热问题的计算

## 4.4 代数方程组的解法

## 4.5 一维非稳态导热问题的计算

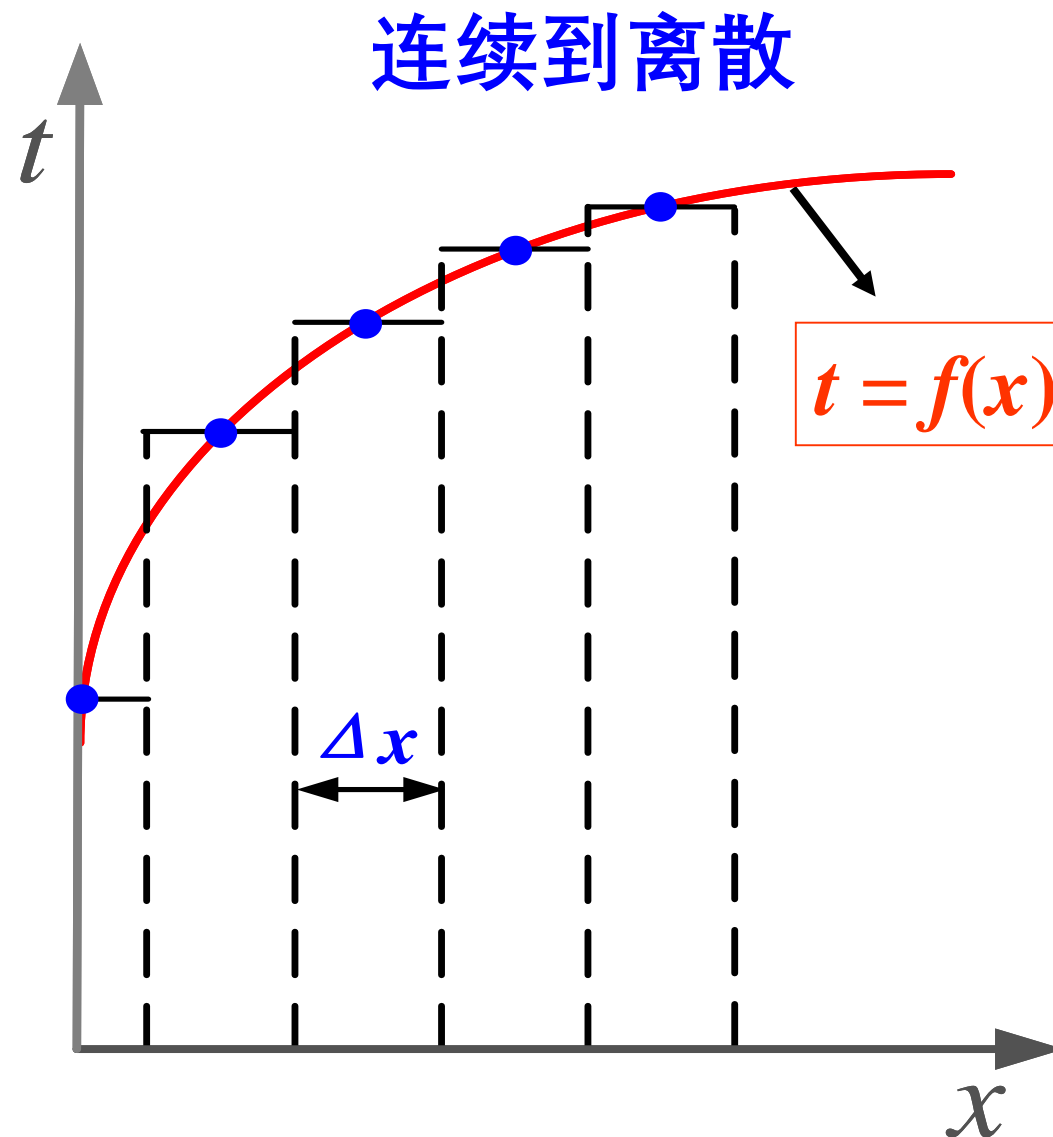
## 4.1 数值求解的基本思想

➤ **分析解**——导热微分方程在定解条件下的积分求解

➤ **数值解**——离散点上被求物理量的值的集合

连续  $\longrightarrow$  离散

微分方程  $\longrightarrow$  代数方程



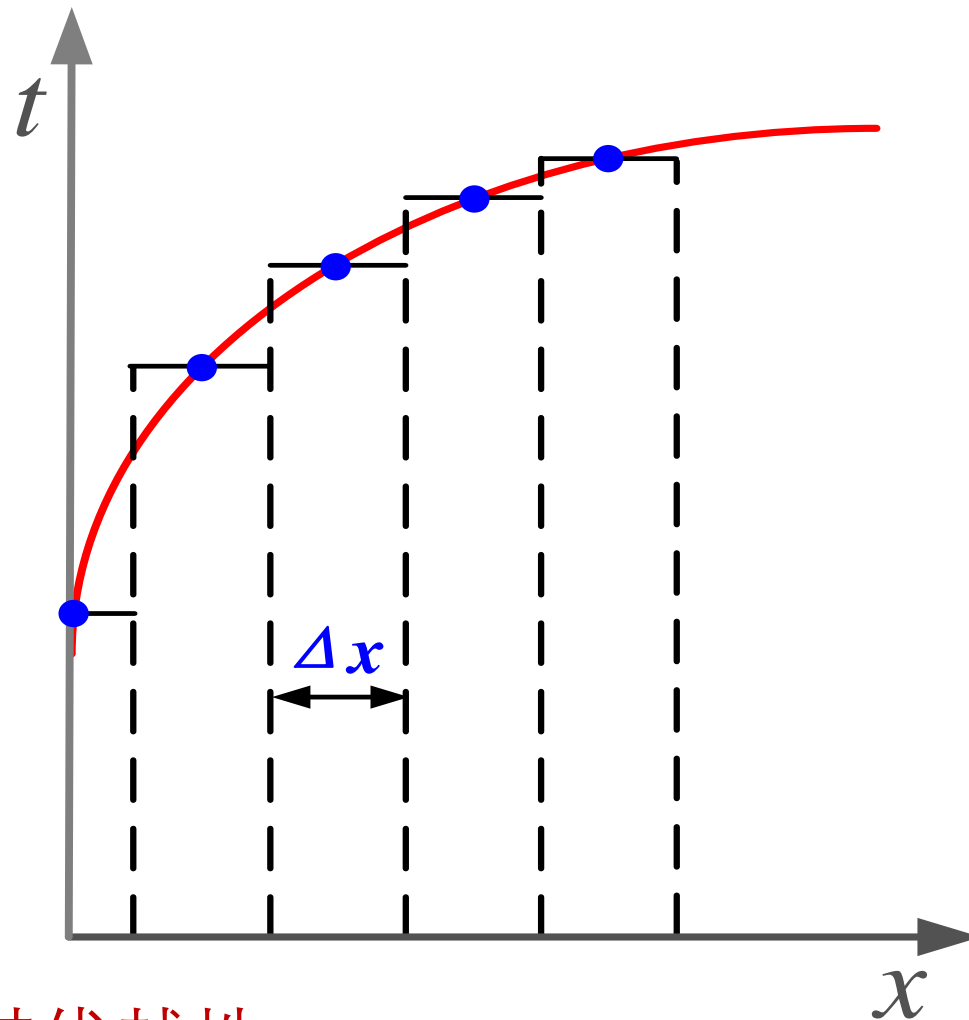
# 分析解和数值解的比较

## ➤ 分析解

- (1) 精确解
- (2) 对复杂问题无法求解
- (3) 具有普遍性

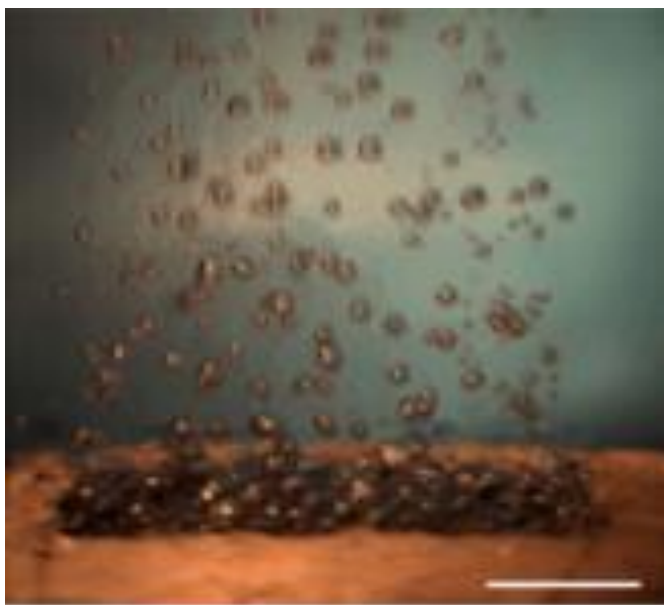
## ➤ 数值解

- (1) 近似解
- (2) 适应性强，特别对于复杂问题更显其优越性
- (3) 与实验法相比成本低
- (4) 具有一定精度

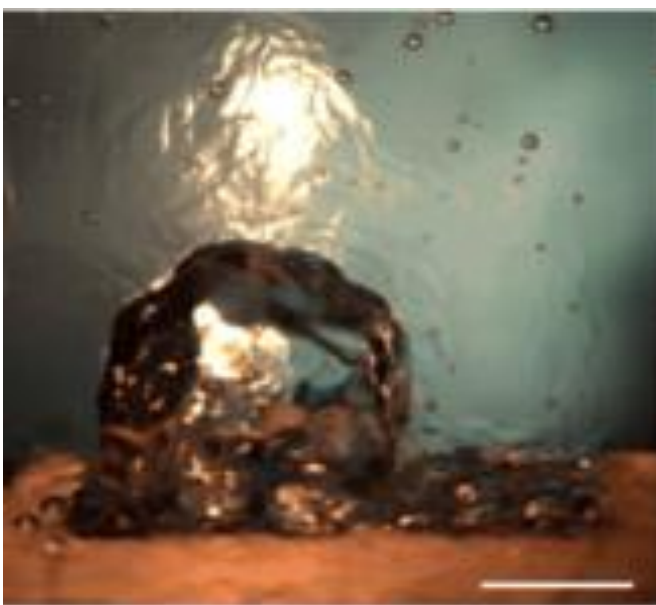


# 实际计算案例

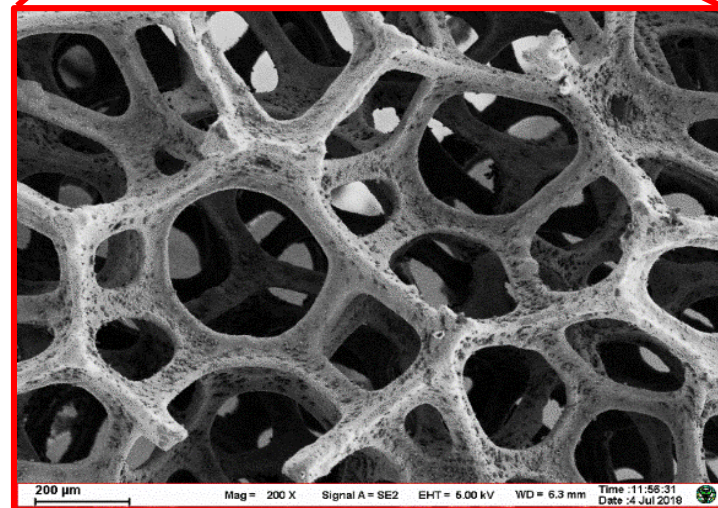
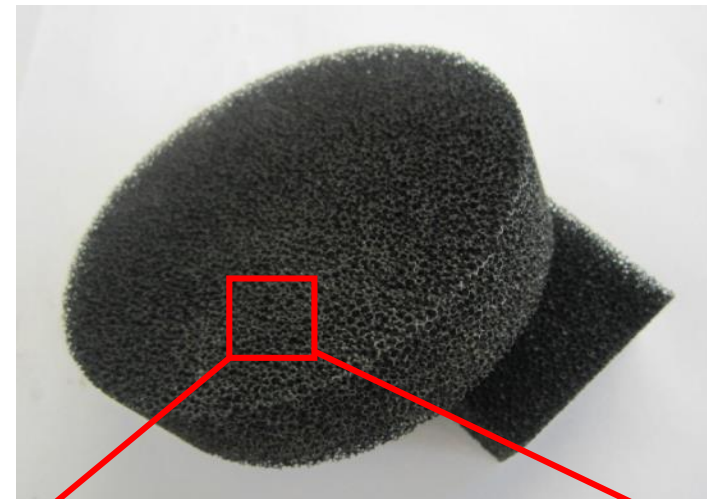
航空航天工程中，航天器大多处于**微重力环境**，高集成度的电子元器件带来较大的散热需求，与常重力条件相比气泡行为变化明显。



-0.714 s  
常重力



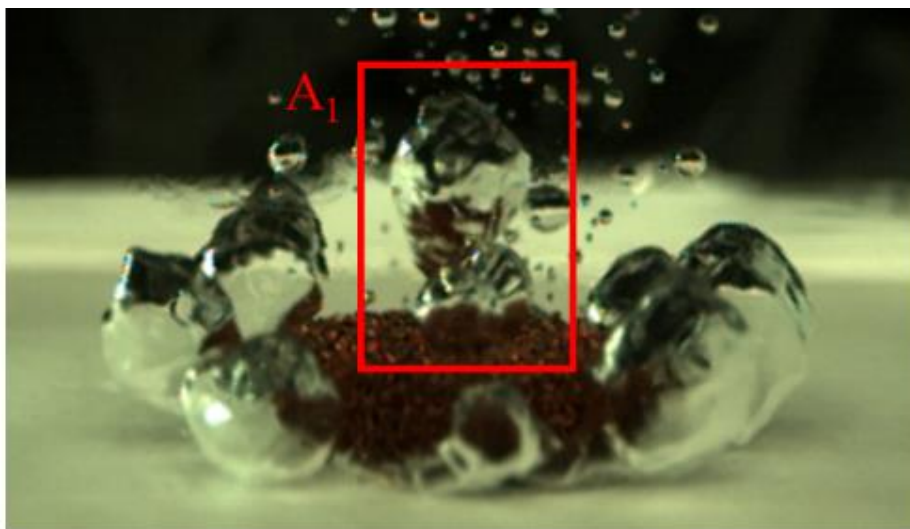
1.074 s  
微重力



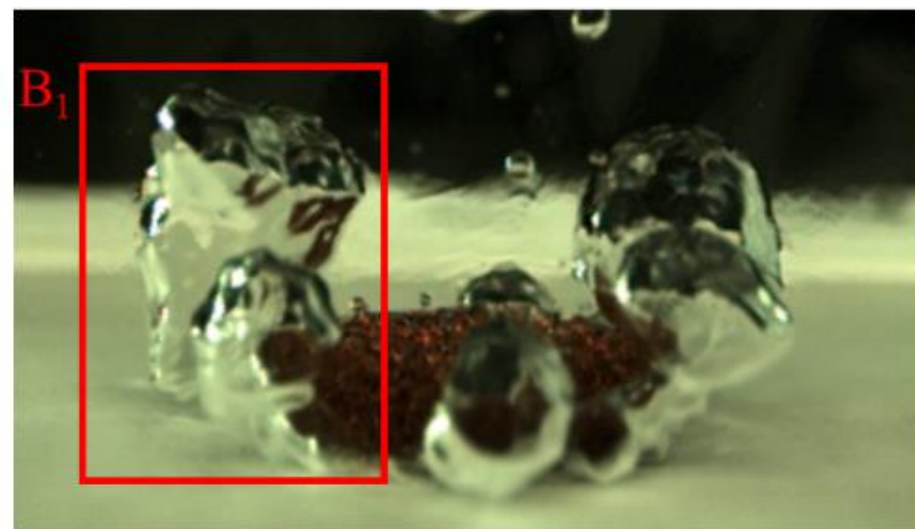
泡沫金属的宏观与微观图像



# 微重力池沸腾模拟

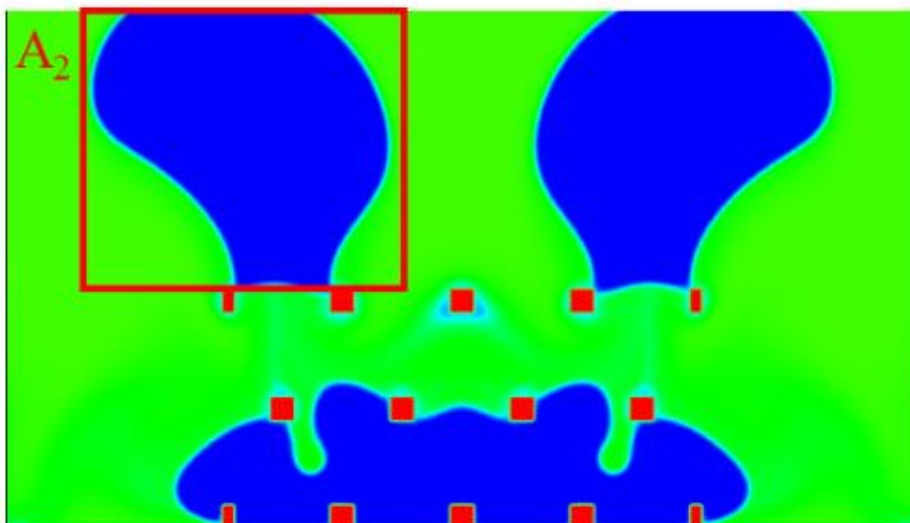


$t' = 0.402s$

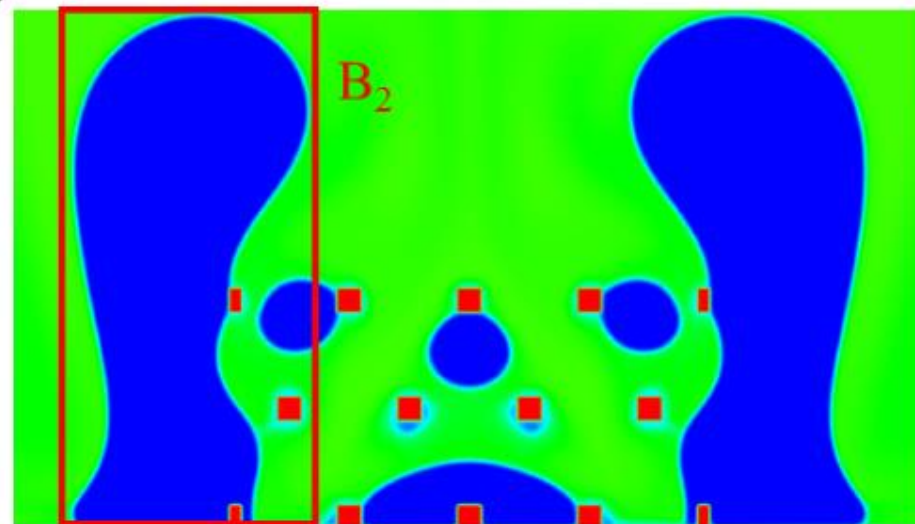


$t' = 1.730s$

(a)



$t = 42000$



$t = 67000$

(b)

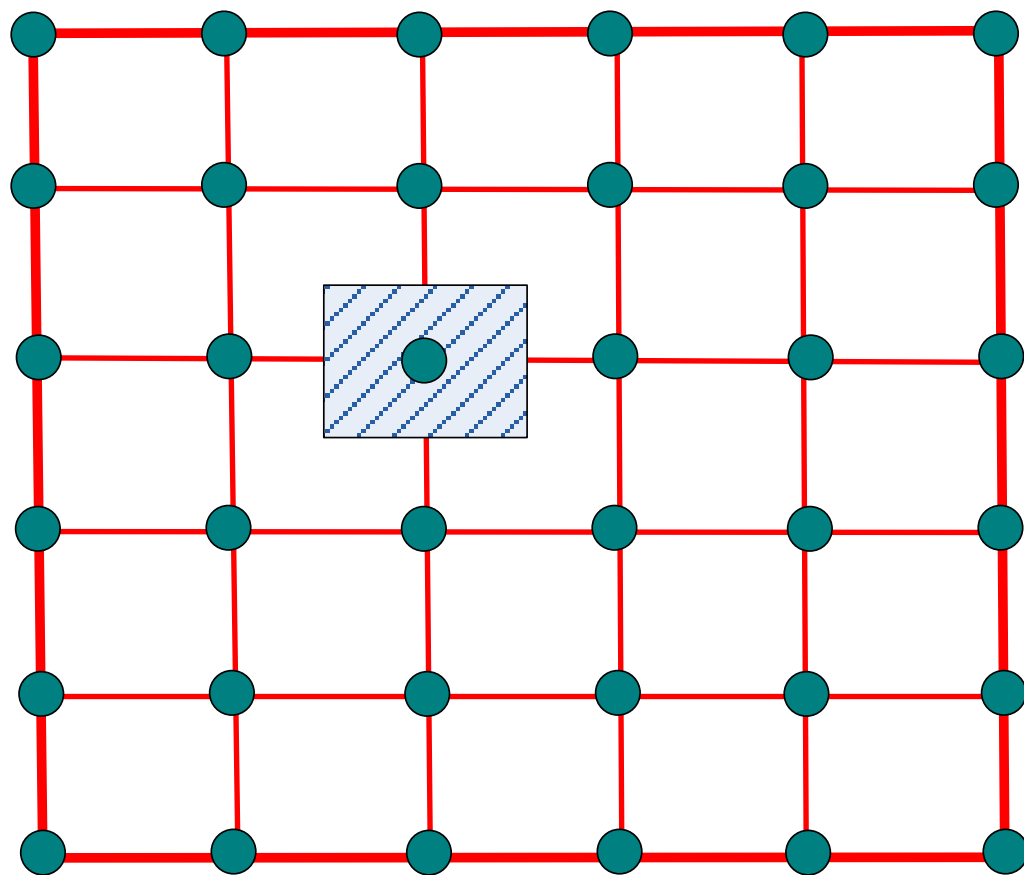
# 常用的数值计算方法

- 有限差分法
- 有限元法
- 有限体积法
- 边界元法
- 等等

# 有限差分法的基本思想

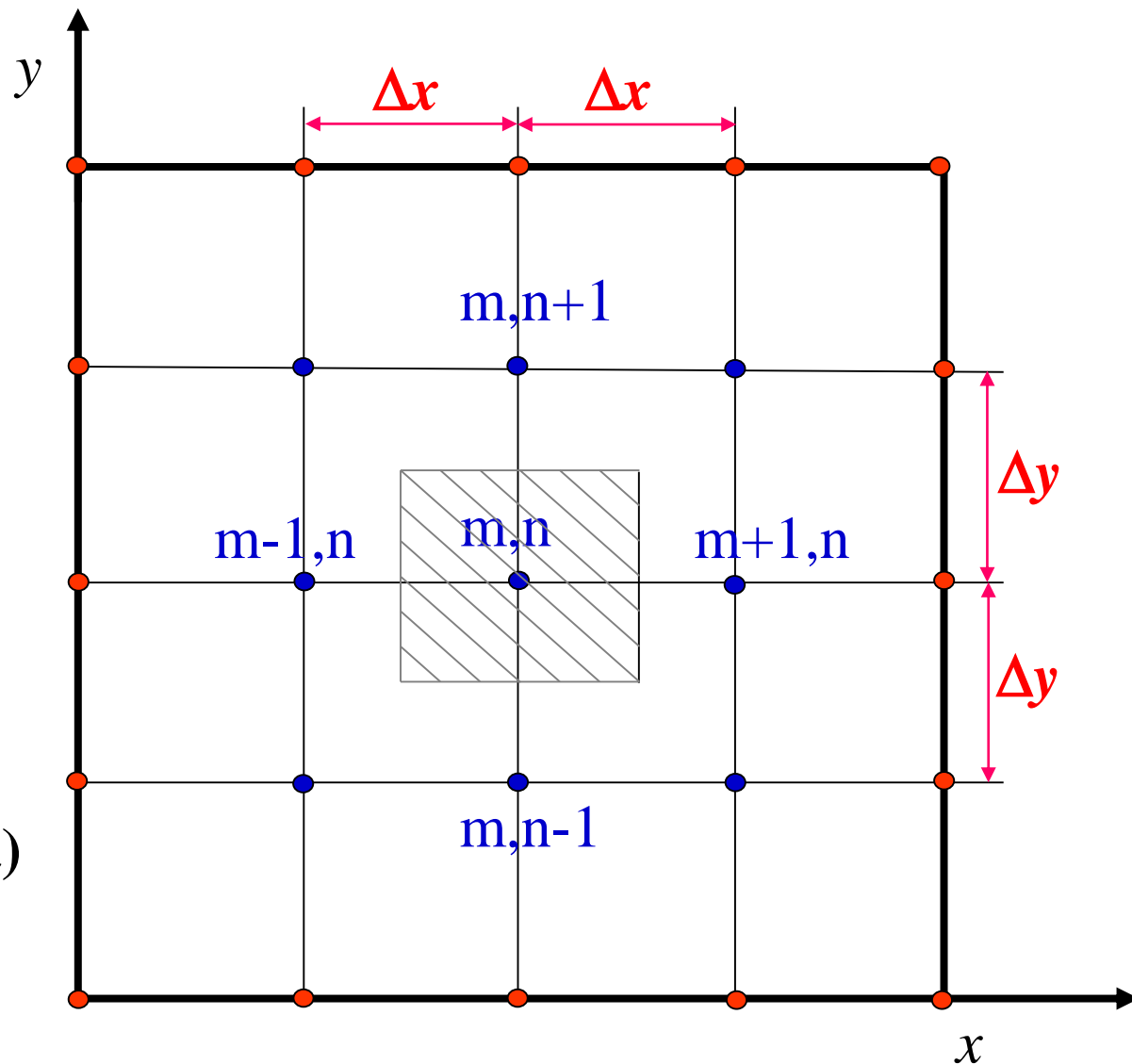
$$\frac{\partial t}{\partial \tau} = \frac{\lambda}{\rho c} \left( \frac{\partial^2 t}{\partial x^2} + \frac{\partial^2 t}{\partial y^2} + \frac{\partial^2 t}{\partial z^2} \right) + \frac{\dot{\Phi}_v}{\rho c}$$

- 用有限个**网格节点的集合**代替连续解；
- 用**有限小的差商**近似代替无限小的**微商（导数）**；
- 用**节点的离散化代数方程（差分方程）**近似代替微分方程；
- 求解差分方程**求取有限节点上的物理量**。



# 离散化方法中的常用术语

- 离散化(Discretization)
- 步长(step length)——即:  $\Delta x$ ,  $\Delta y$
- 节点(Nodes)——边界节点, 内节点
- 网格(Grids)
- 控制容积(control volume)或单元体(element)



# 网格划分演示

# MATLAB®

*The Language of Technical Computing*

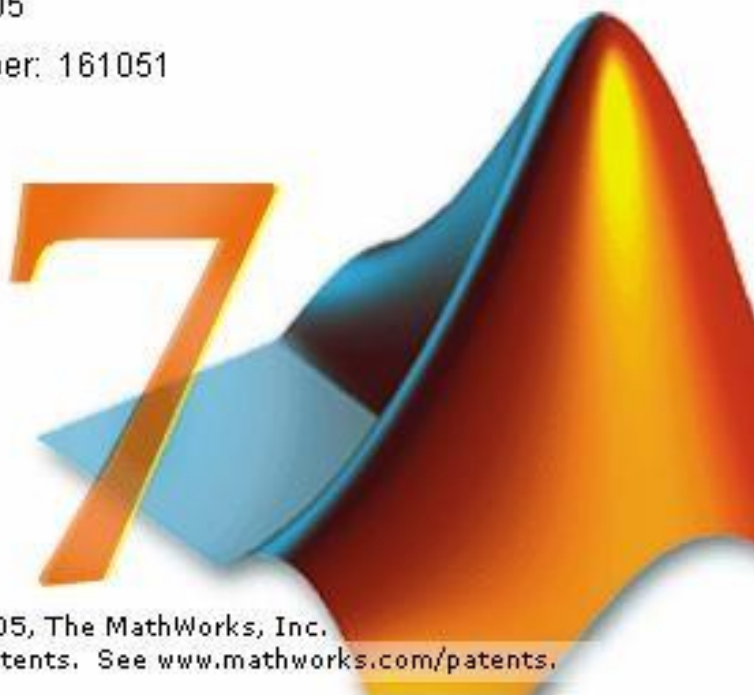
Version 7.1.0.246 (R14) Service Pack 3

August 02, 2005

License Number: 161051

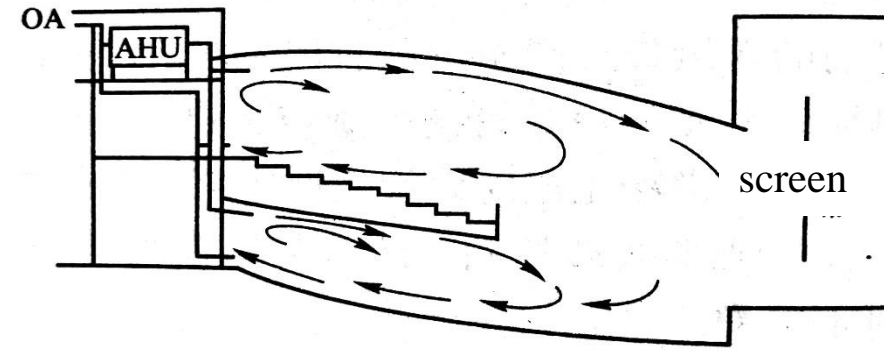
ZhangHui

Powernet

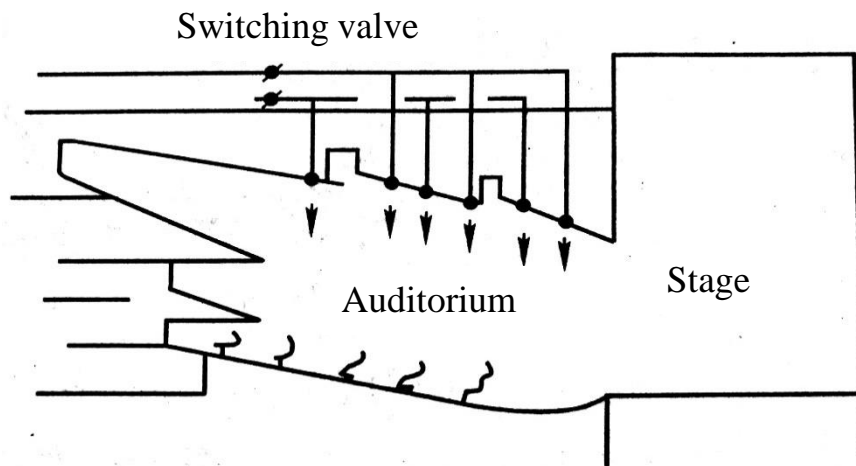


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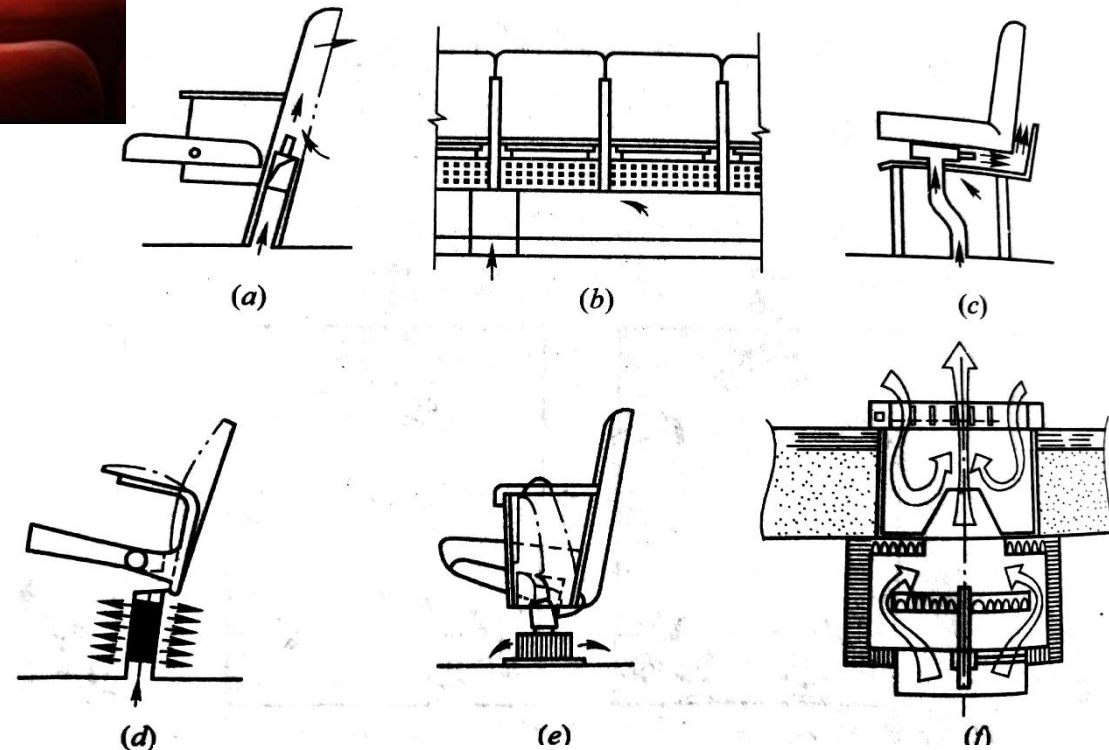
# Radiation cooling technology in Theater (剧场辐射制冷技术)



Nozzle deliver model

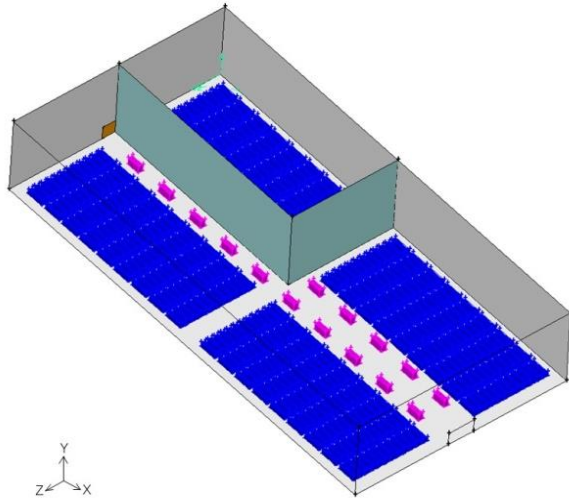


Top-supply and down-return air supply model

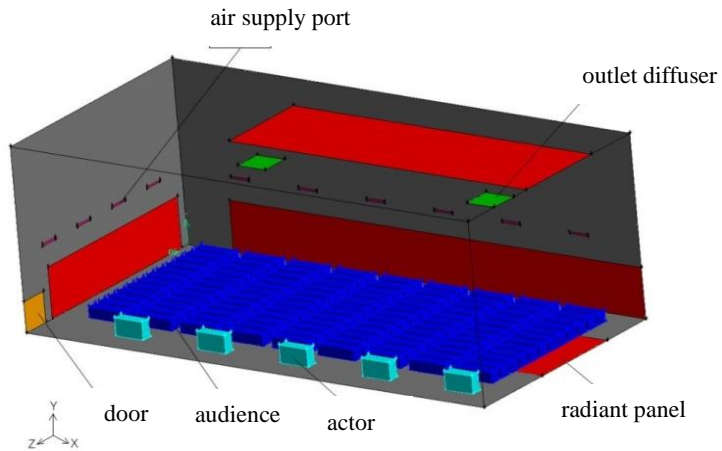




## ■ Establishment of heat transfer model

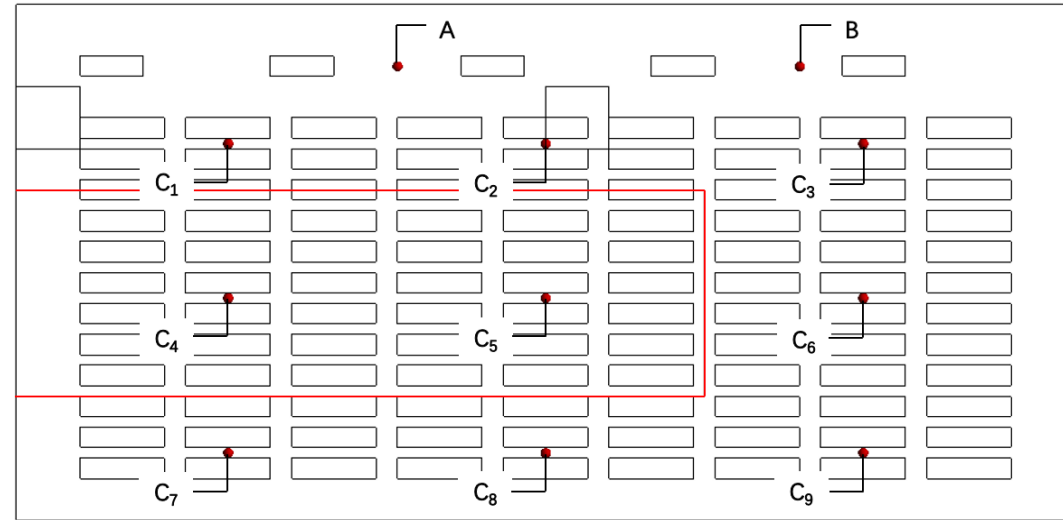


Stereogram of the theater building

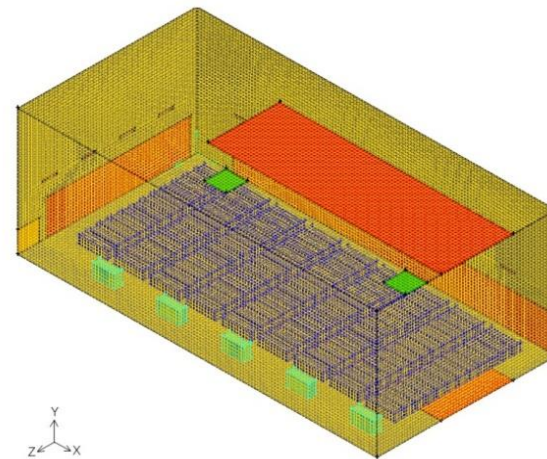


Physical model diagram of the theater

## ■ Setting of indoor monitoring point



- Audience area : C1~C9 nine monitoring points
- Performance area : A、B two monitoring points
- The inner area of the red line is where the radiant panel is placed on the ground.

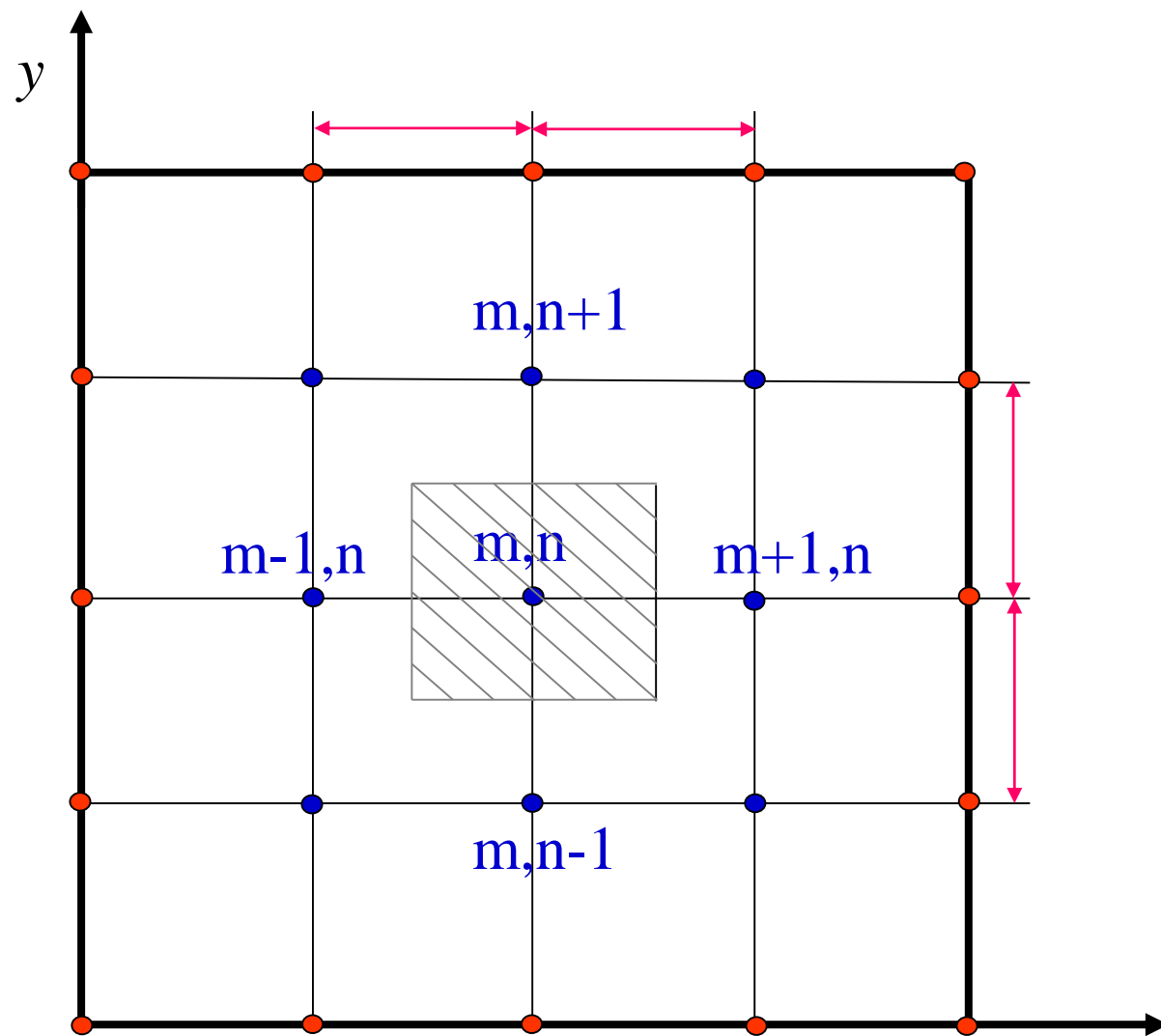


Mesh generation

Meshing under the premise of meeting the accuracy of the subject

**Number of grids is  
2034283**

- 控制容积大小取决于节点数；
- 网格越细密，节点越多，计算时间越长，结果越接近分析解

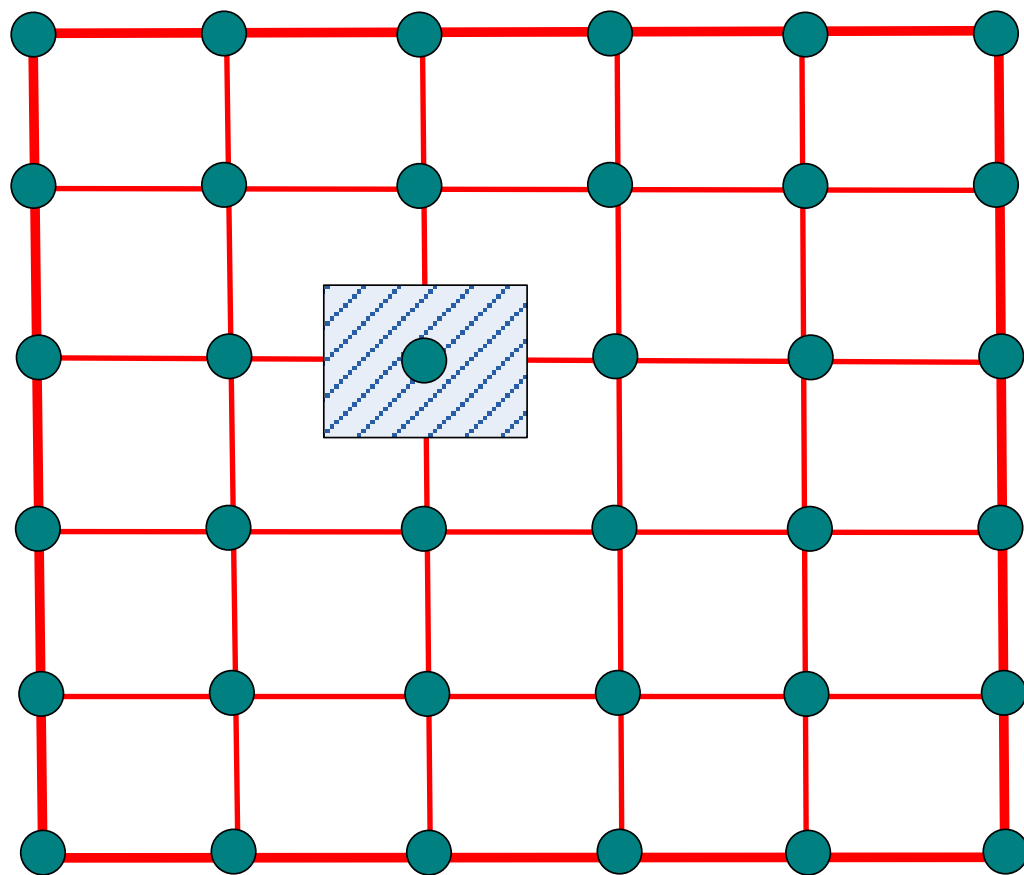




# 有限差分法的基本思想

$$\frac{\partial t}{\partial \tau} = \frac{\lambda}{\rho c} \left( \frac{\partial^2 t}{\partial x^2} + \frac{\partial^2 t}{\partial y^2} + \frac{\partial^2 t}{\partial z^2} \right) + \frac{\dot{\Phi}_v}{\rho c}$$

- 用有限个**网格节点的集合**代替连续解；
- 用**有限小的差商**近似代替无限小的**微商（导数）**；
- 用**节点的离散化代数方程（差分方程）**近似代替微分方程；
- 求解差分方程**求取有限节点上的物理量**。



# 四种节点差分格式

微商形式 (a)

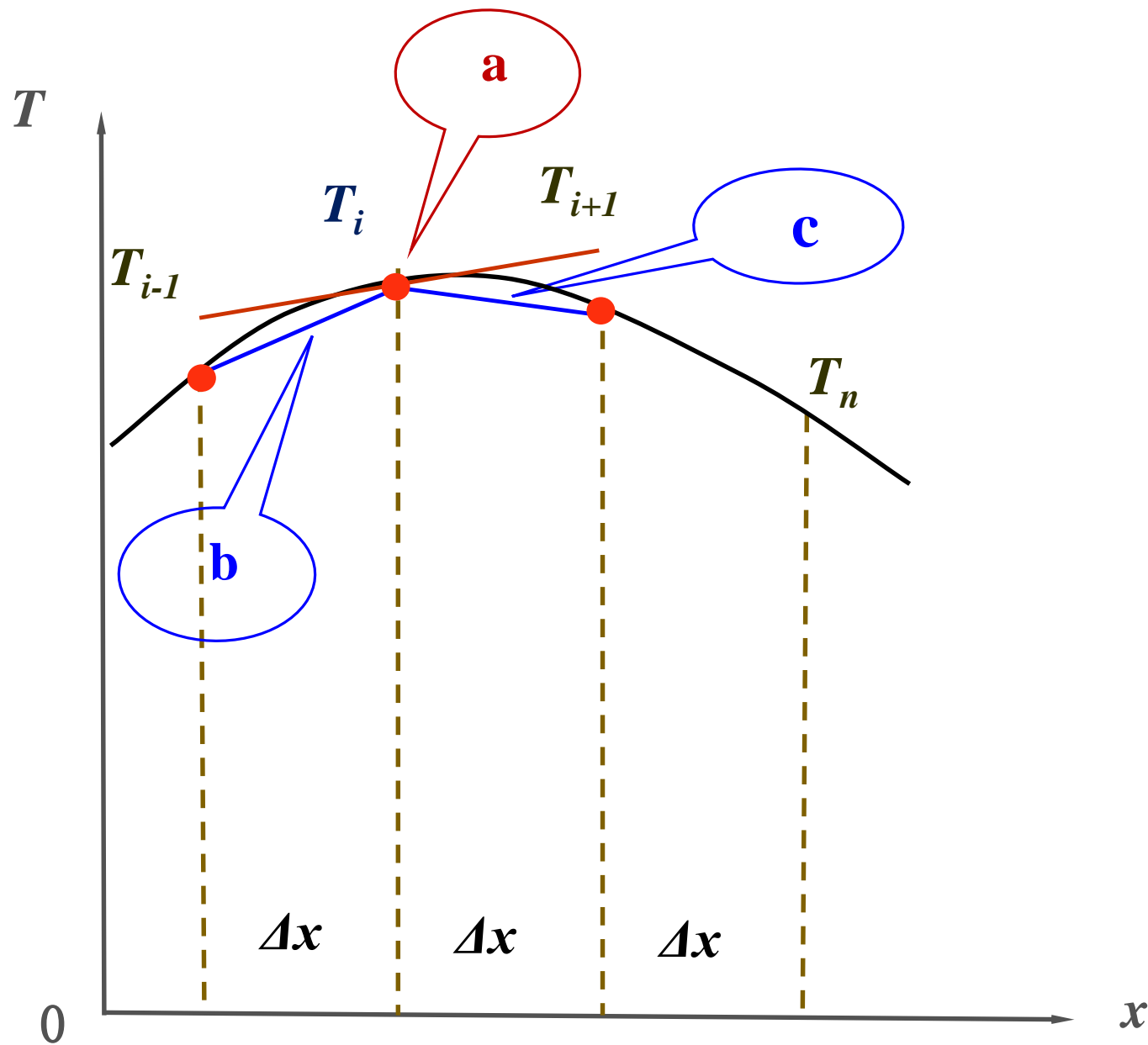
$$\frac{dT_i}{dx} = \lim_{\Delta x \rightarrow 0} \frac{\Delta T_i}{\Delta x}$$

1) 向后差分格式(b:  $T_i \rightarrow T_{i-1}$ )

$$\frac{dT_i}{dx} \approx \frac{T(x_i) - T(x_{i-1})}{\Delta x}$$

2) 向前差分格式(c:  $T_i \rightarrow T_{i+1}$ )

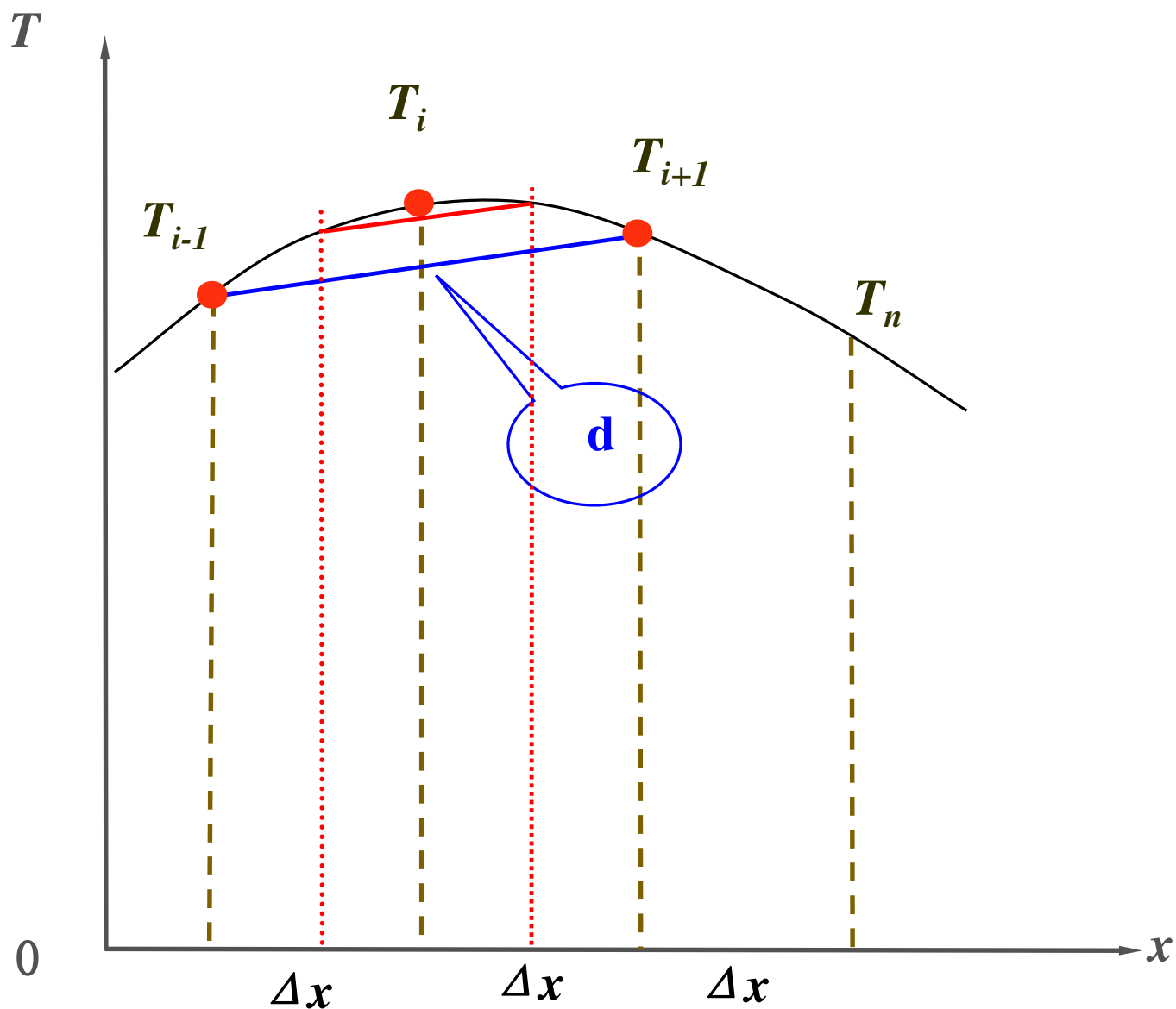
$$\frac{dT_i}{dx} \approx \frac{T(x_{i+1}) - T(x_i)}{\Delta x}$$



### 3) 中心差分格式(d)

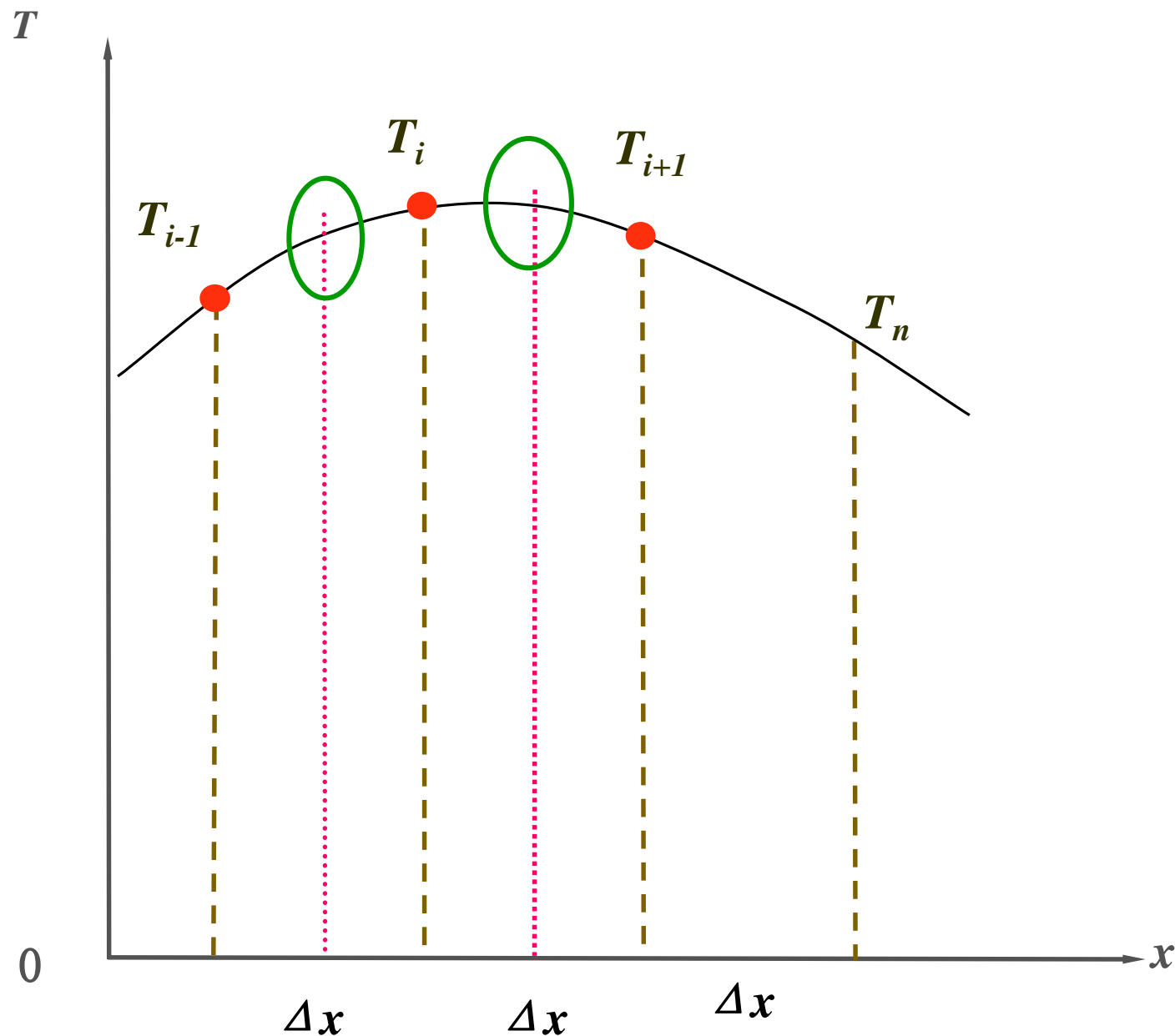
$$\frac{dT_i}{dx} \approx \frac{T(x_{i+1}) - T(x_{i-1}))}{2\Delta x}$$

$$\frac{dT_i}{dx} \approx \frac{T(x_{i+1/2}) - T(x_{i-1/2}))}{\Delta x}$$

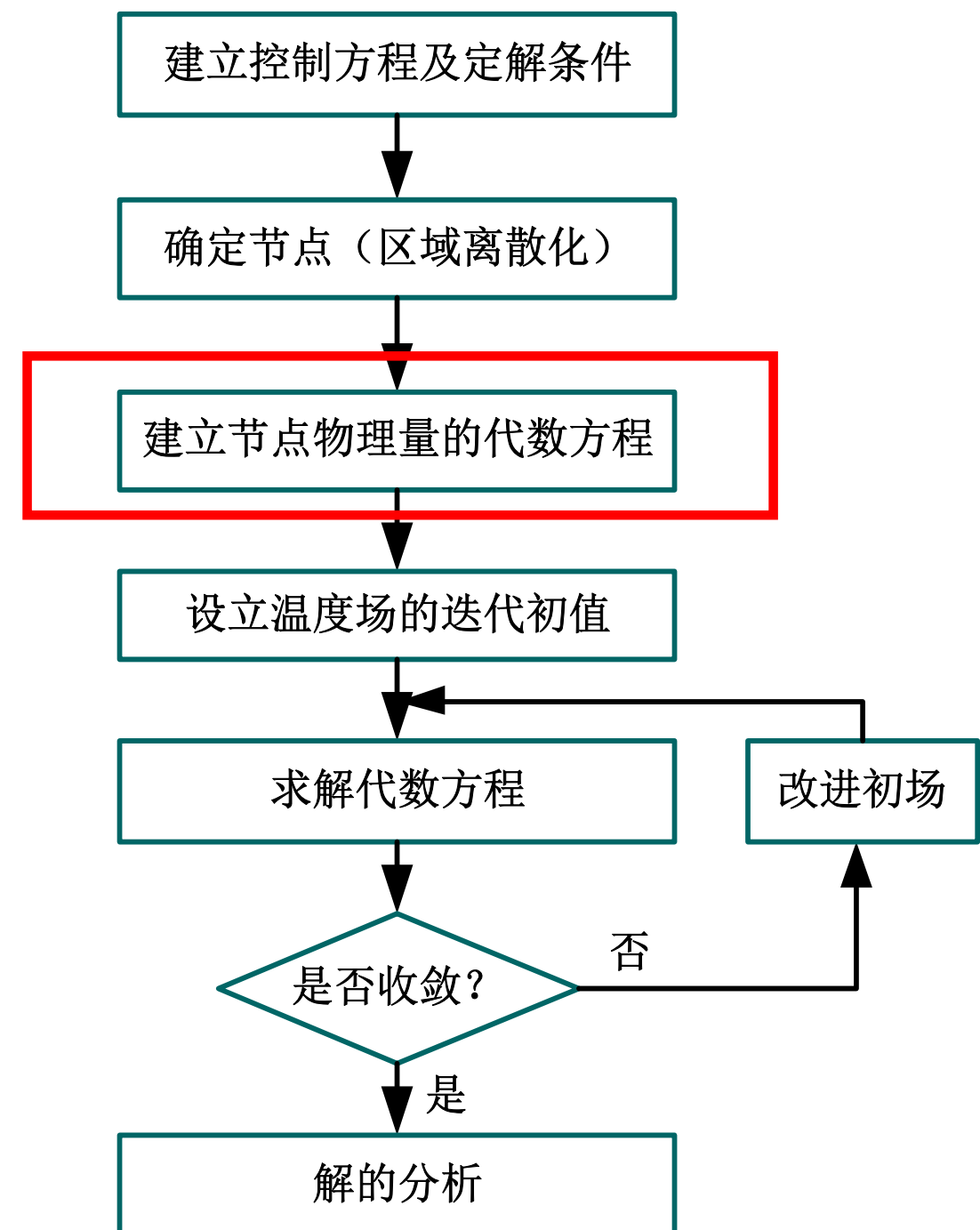
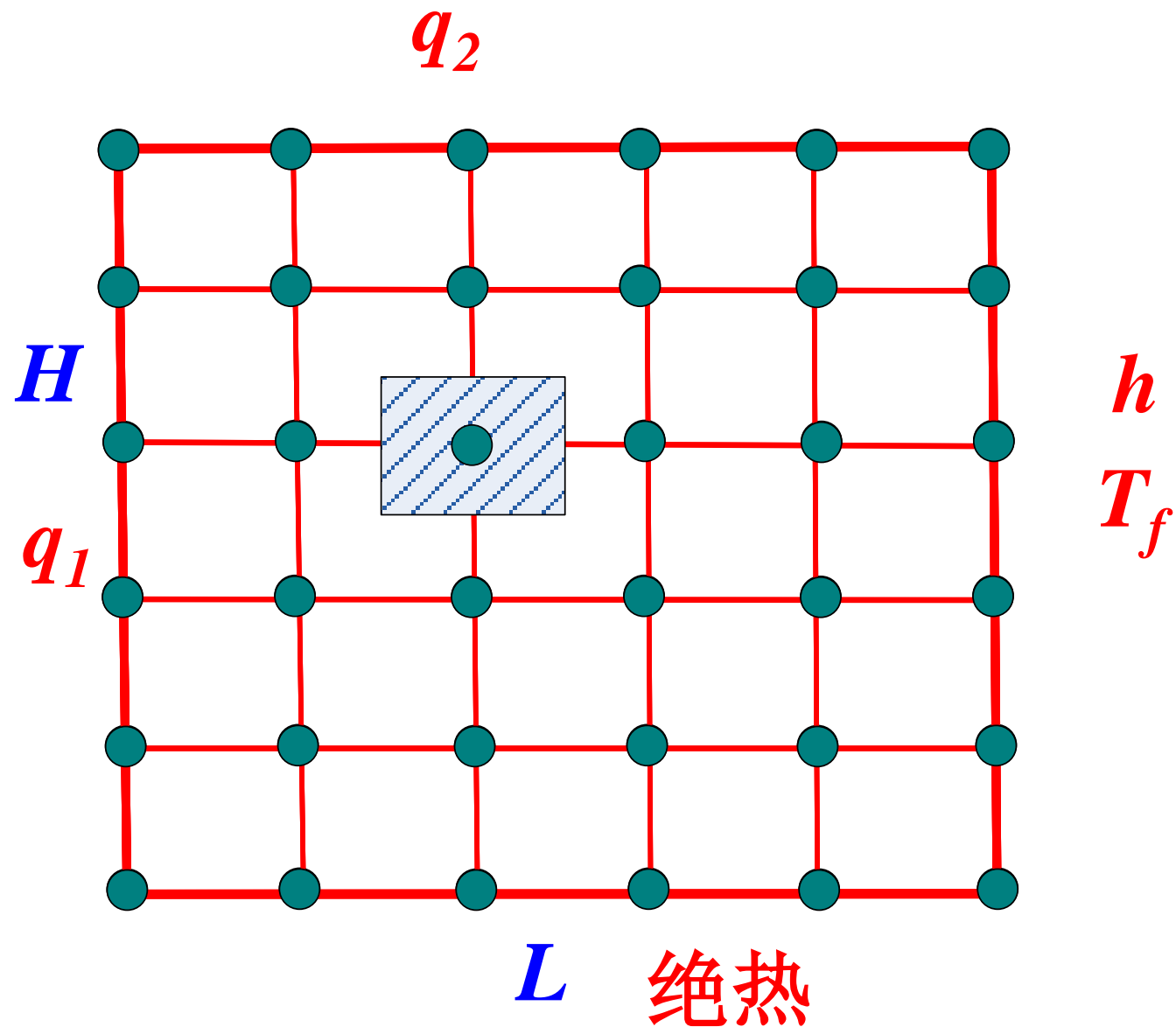


#### 4) 二阶微商（导数）的差分格式

$$\begin{aligned}\frac{d^2 T_i}{dx^2} &= \frac{d}{dx} \left( \frac{dT_i}{dx} \right) \\ &\approx \frac{\left( \frac{dT}{dx} \right)_{x=x_i+1/2} - \left( \frac{dT}{dx} \right)_{x=x_i-1/2}}{\Delta x} \\ &\approx \frac{\frac{T(x_{i+1}) - T(x_i)}{\Delta x} - \frac{T(x_i) - T(x_{i-1}))}{\Delta x}}{\Delta x} \\ &\approx \frac{T(x_{i+1}) - 2T(x_i) + T(x_{i-1}))}{(\Delta x)^2}\end{aligned}$$



# 导热问题数值求解基本步骤



4.1 数值求解的基本思想

4.2 离散方程的建立方法

4.3 二维稳态导热问题的计算

4.4 代数方程组的解法

4.5 一维非稳态导热问题的计算

## 4.2 离散方程的建立方法

- (1) 控制容积热平衡法：着重于从物理的观点来分析
- (2) Taylor（泰勒）级数展开法：偏重于从数学的角度进行推导
- (3) 多项式拟合法
- (4) 控制容积积分法

# (1) 热平衡法

## 基本思想

从基本物理现象和基本定律出发  
依据能量守恒和傅立叶导热定律建立方程

## 能量守恒

从所有方向流入控制体的总热流量



控制体内热源生成热  $\equiv$  控制体内能的增量

单位: [W]

$$\Phi_i + (-\Phi_o) + \Phi_v = \Phi_\tau$$



# 热平衡法

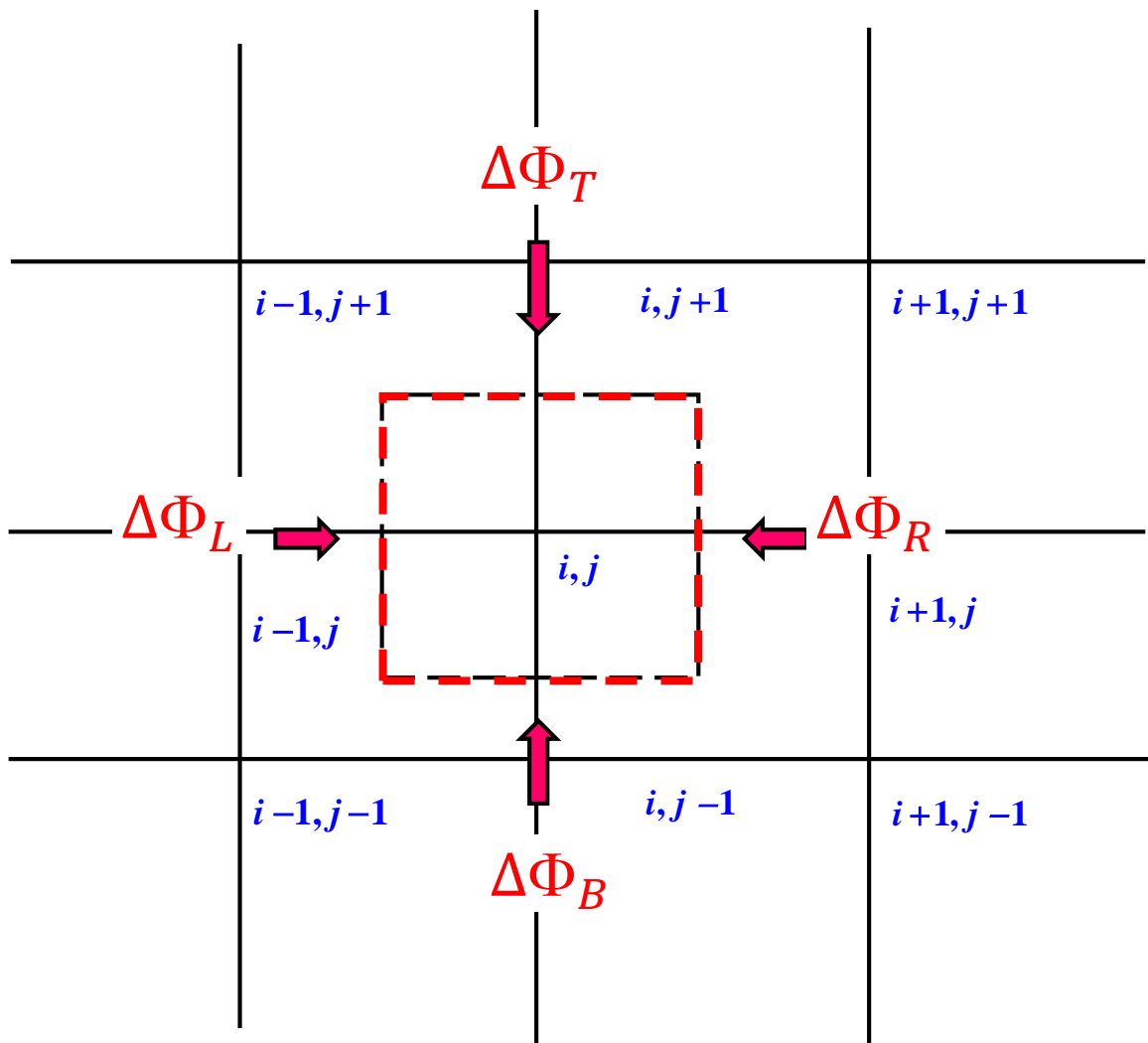
$$\Phi_i + (-\Phi_o) + \cancel{\Phi_v} = \cancel{\Phi_\tau}$$

稳态无内热源条件

从所有方向导入控制体的总热流量=0

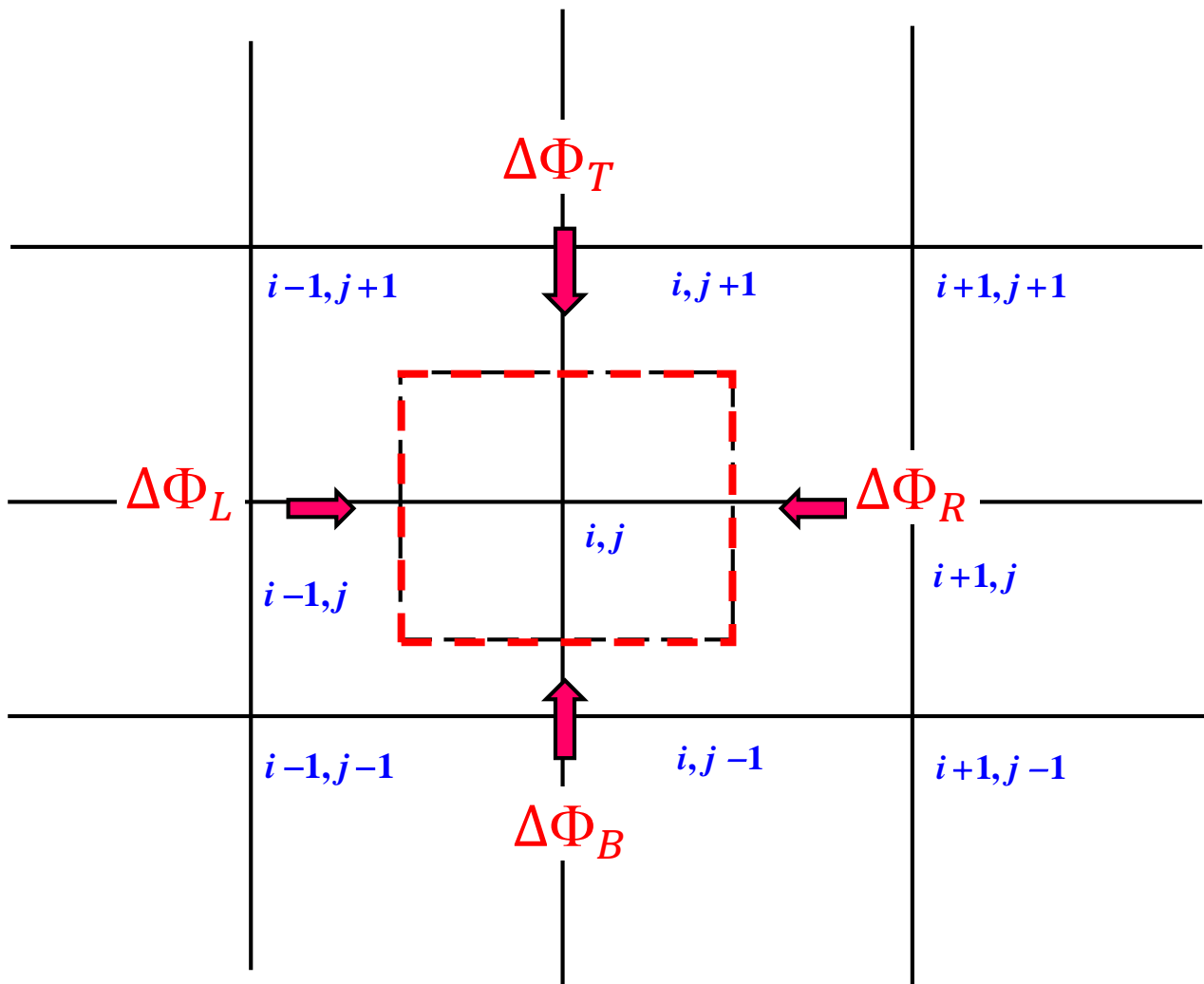
能量平衡方程

$$\Delta\Phi_L + \Delta\Phi_R + \Delta\Phi_B + \Delta\Phi_T = 0$$



## 能量平衡方程

$$\Delta\Phi_L + \Delta\Phi_R + \Delta\Phi_B + \Delta\Phi_T = 0$$



$$\Delta\Phi_L = q_L \Delta y = -\lambda \left. \frac{\partial t}{\partial x} \right|_L \Delta y$$

$$\Delta\Phi_R = q_R \Delta y = \lambda \left. \frac{\partial t}{\partial x} \right|_R \Delta y$$

$$\Delta\Phi_B = q_B \Delta x = -\lambda \left. \frac{\partial t}{\partial y} \right|_B \Delta x$$

$$\Delta\Phi_T = q_T \Delta x = \lambda \left. \frac{\partial t}{\partial y} \right|_T \Delta x$$

# 一阶导数的近似——差商

## 用差商代替微商（导数）

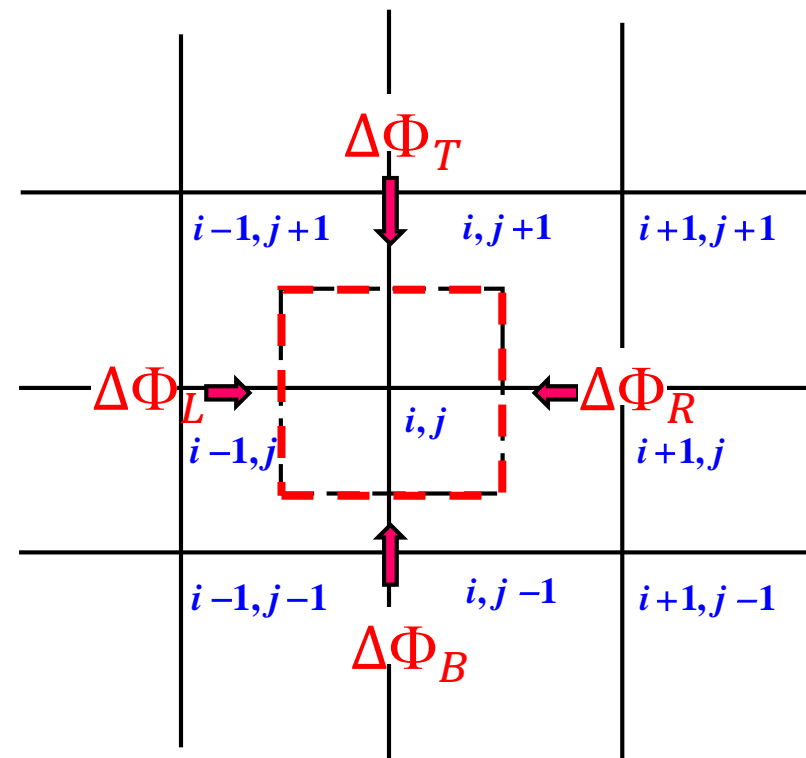
$$\begin{aligned}\left. \frac{\partial t}{\partial x} \right|_L &\approx \frac{t_{i,j} - t_{i-1,j}}{\Delta x} & \left. \frac{\partial t}{\partial x} \right|_R &\approx \frac{t_{i+1,j} - t_{i,j}}{\Delta x} \\ \left. \frac{\partial t}{\partial y} \right|_B &\approx \frac{t_{i,j} - t_{i,j-1}}{\Delta y} & \left. \frac{\partial t}{\partial y} \right|_T &\approx \frac{t_{i,j+1} - t_{i,j}}{\Delta y}\end{aligned}$$

$$\Delta\Phi_L = q_L \Delta y = -\lambda \left. \frac{\partial t}{\partial x} \right|_L \Delta y$$

$$\Delta\Phi_R = q_R \Delta y = \lambda \left. \frac{\partial t}{\partial x} \right|_R \Delta y$$

$$\Delta\Phi_B = q_B \Delta x = -\lambda \left. \frac{\partial t}{\partial y} \right|_B \Delta x$$

$$\Delta\Phi_T = q_T \Delta x = \lambda \left. \frac{\partial t}{\partial y} \right|_T \Delta x$$



$$\Delta\Phi_L = q_L \Delta y = -\lambda \left. \frac{\partial t}{\partial x} \right|_L \Delta y$$

$$\Delta\Phi_R = q_R \Delta y = \lambda \left. \frac{\partial t}{\partial x} \right|_R \Delta y$$

$$\Delta\Phi_B = q_B \Delta y = -\lambda \left. \frac{\partial t}{\partial y} \right|_B \Delta x$$

$$\Delta\Phi_T = q_T \Delta y = \lambda \left. \frac{\partial t}{\partial y} \right|_T \Delta x$$

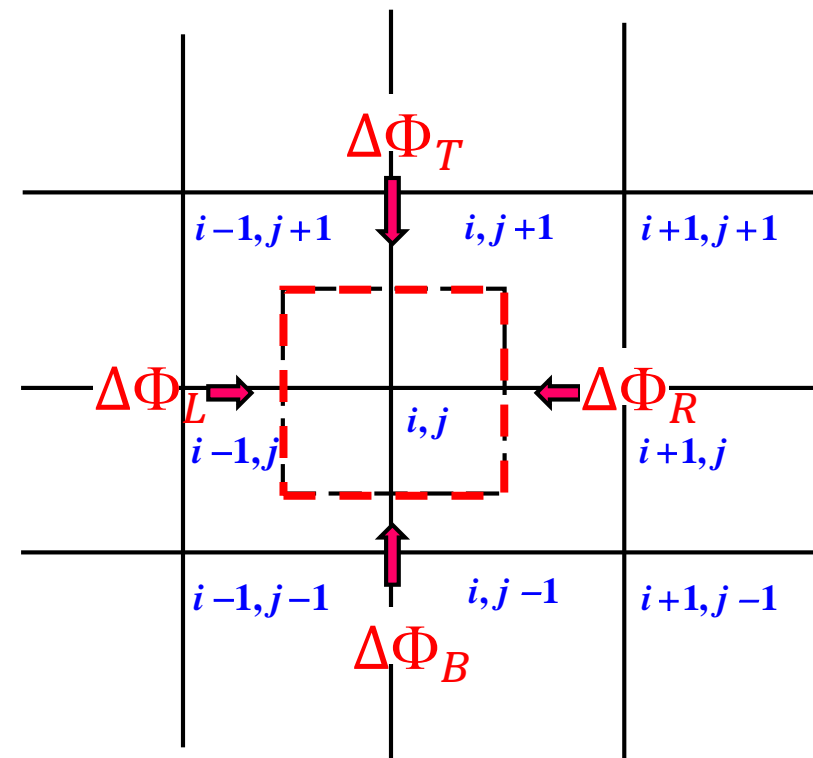
$$\left. \frac{\partial t}{\partial x} \right|_L \approx \frac{t_{i,j} - t_{i-1,j}}{\Delta x}$$

$$\left. \frac{\partial t}{\partial x} \right|_R \approx \frac{t_{i+1,j} - t_{i,j}}{\Delta x}$$

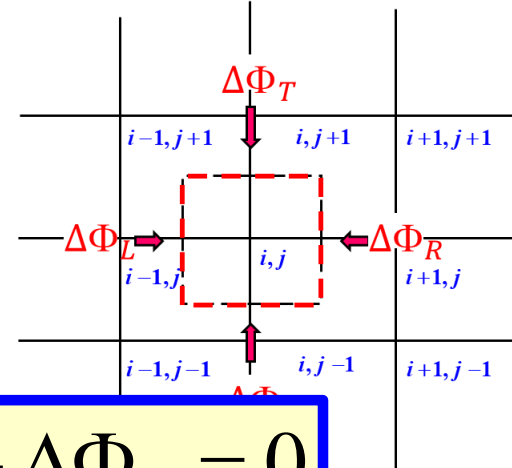
$$\left. \frac{\partial t}{\partial y} \right|_B \approx \frac{t_{i,j} - t_{i,j-1}}{\Delta y}$$

$$\left. \frac{\partial t}{\partial y} \right|_T \approx \frac{t_{i,j+1} - t_{i,j}}{\Delta y}$$

$$\left\{ \begin{array}{ll} \Delta\Phi_L = -\lambda \Delta y \left( \frac{t_{i,j} - t_{i-1,j}}{\Delta x} \right), & \Delta\Phi_R = \lambda \Delta y \left( \frac{t_{i+1,j} - t_{i,j}}{\Delta x} \right) \\ \Delta\Phi_B = -\lambda \Delta x \left( \frac{t_{i,j} - t_{i,j-1}}{\Delta y} \right), & \Delta\Phi_T = \lambda \Delta x \left( \frac{t_{i,j+1} - t_{i,j}}{\Delta y} \right) \end{array} \right.$$



# 微元体能量守恒的近似表达



$$\Delta\Phi_L = -\lambda\Delta y \left( \frac{t_{i,j} - t_{i-1,j}}{\Delta x} \right), \quad \Delta\Phi_R = \lambda\Delta y \left( \frac{t_{i+1,j} - t_{i,j}}{\Delta x} \right)$$

$$\Delta\Phi_B = -\lambda\Delta x \left( \frac{t_{i,j} - t_{i,j-1}}{\Delta y} \right), \quad \Delta\Phi_T = \lambda\Delta x \left( \frac{t_{i,j+1} - t_{i,j}}{\Delta y} \right)$$

$$\Delta\Phi_L + \Delta\Phi_R + \Delta\Phi_B + \Delta\Phi_T = 0$$

$$-\cancel{\lambda\Delta y} \left( \frac{t_{i,j} - t_{i-1,j}}{\cancel{\Delta x}} \right) + \cancel{\lambda\Delta y} \left( \frac{t_{i+1,j} - t_{i,j}}{\cancel{\Delta x}} \right) - \cancel{\lambda\Delta x} \left( \frac{t_{i,j} - t_{i,j-1}}{\cancel{\Delta y}} \right) + \cancel{\lambda\Delta x} \left( \frac{t_{i,j+1} - t_{i,j}}{\cancel{\Delta y}} \right) = 0$$

$$\Delta x = \Delta y$$

$$-(t_{i,j} - t_{i-1,j}) + (t_{i+1,j} - t_{i,j}) - (t_{i,j} - t_{i,j-1}) + (t_{i,j+1} - t_{i,j}) = 0$$



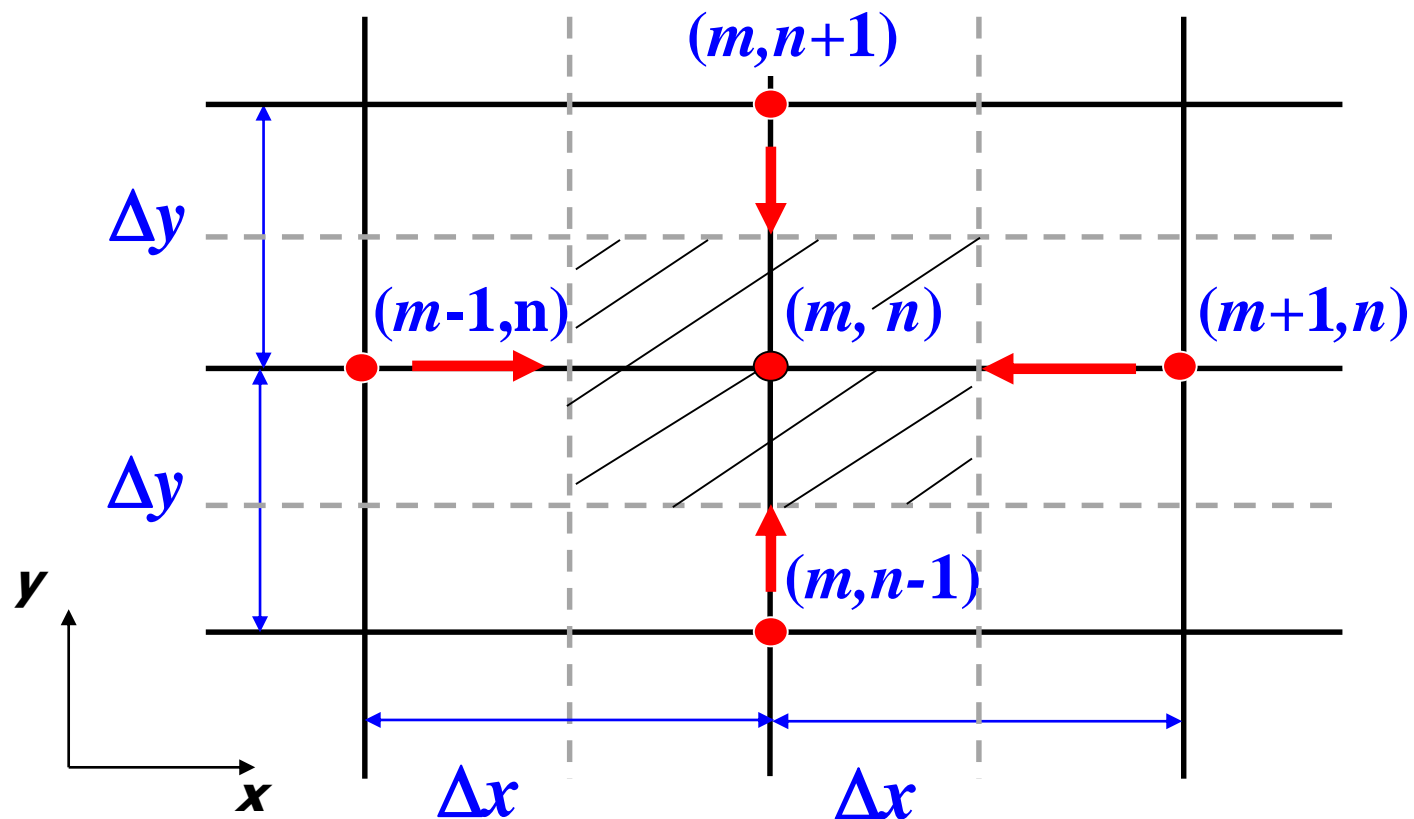
$$t_{i,j} = \frac{1}{4} (t_{i-1,j} + t_{i+1,j} + t_{i,j-1} + t_{i,j+1})$$

## (2) Taylor (泰勒) 级数展开法

函数的泰勒级数展开式为

$$f(x+dx) = f(x) + \frac{df(x)}{dx} dx + \frac{1}{2!} \frac{d^2 f(x)}{dx^2} dx^2 + \frac{1}{3!} \frac{d^3 f(x)}{dx^3} dx^3 + \dots$$

对相邻节点 $(m+1,n)$ 及 $(m-1,n)$   
分别写出温度对节点 $(m,n)$  的  
泰勒级数展开式



$$f(x+dx) = f(x) + \frac{df(x)}{dx} dx + \frac{1}{2!} \frac{d^2 f(x)}{dx^2} dx^2 + \frac{1}{3!} \frac{d^3 f(x)}{dx^3} dx^3 + \dots$$

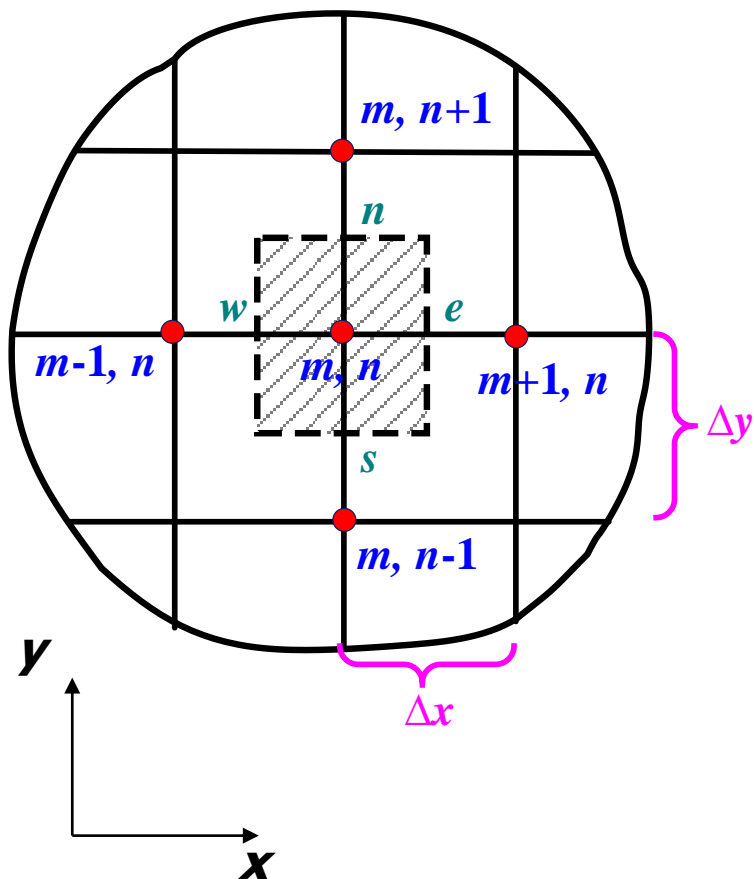
(1) 对相邻节点写出温度 $t$  对内节点 $(m, n)$  的泰勒级数展开式

➤ 用  $t_{m,n}$  来表示  $t_{m+1,n}$

$$t_{m+1,n} = t_{m,n} + \left. \frac{\partial t}{\partial x} \right|_{m,n} \Delta x + \left. \frac{\partial^2 t}{\partial x^2} \right|_{m,n} \frac{\Delta x^2}{2!} + \left. \frac{\partial^3 t}{\partial x^3} \right|_{m,n} \frac{\Delta x^3}{3!} + \dots$$

➤ 用  $t_{m,n}$  来表示  $t_{m-1,n}$

$$t_{m-1,n} = t_{m,n} - \left. \frac{\partial t}{\partial x} \right|_{m,n} \Delta x + \left. \frac{\partial^2 t}{\partial x^2} \right|_{m,n} \frac{\Delta x^2}{2!} - \left. \frac{\partial^3 t}{\partial x^3} \right|_{m,n} \frac{\Delta x^3}{3!} + \dots$$



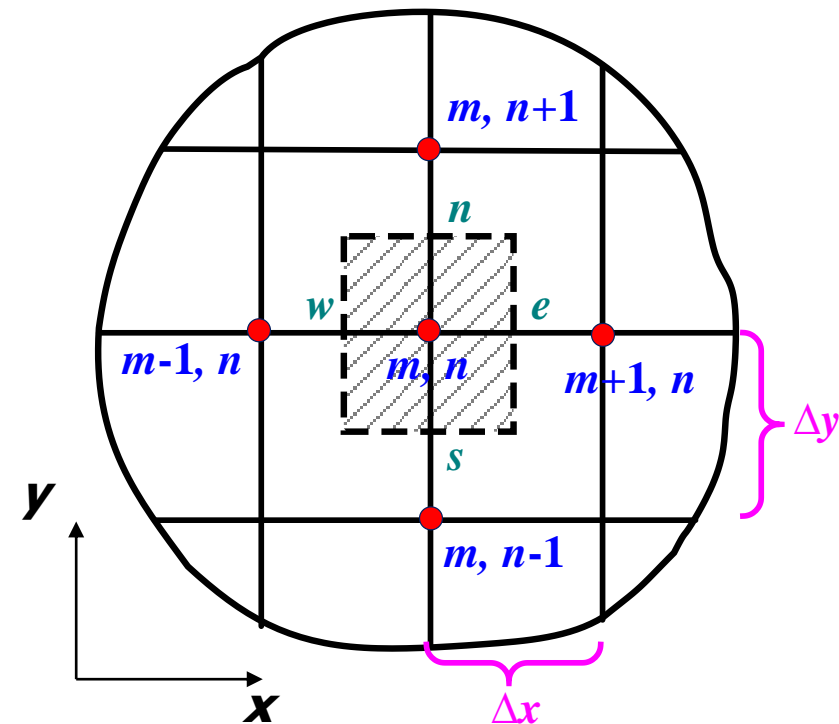
# 内节点离散方程

$$t_{m+1,n} = t_{m,n} + \frac{\partial t}{\partial x} \bigg|_{m,n} \Delta x + \frac{\partial^2 t}{\partial x^2} \bigg|_{m,n} \frac{\Delta x^2}{2!} + \frac{\partial^3 t}{\partial x^3} \bigg|_{m,n} \frac{\Delta x^3}{3!} + \dots$$

$$t_{m-1,n} = t_{m,n} - \frac{\partial t}{\partial x} \bigg|_{m,n} \Delta x + \frac{\partial^2 t}{\partial x^2} \bigg|_{m,n} \frac{\Delta x^2}{2!} - \frac{\partial^3 t}{\partial x^3} \bigg|_{m,n} \frac{\Delta x^3}{3!} + \dots$$

$$t_{m+1,n} + t_{m-1,n} = 2t_{m,n} + \Delta x^2 \frac{\partial^2 t}{\partial x^2} \bigg|_{m,n} + \frac{\Delta x^4}{12} \frac{\partial^4 t}{\partial x^4} \bigg|_{m,n} + \dots$$

$$\frac{\partial^2 t}{\partial x^2} \bigg|_{m,n} = \frac{t_{m+1,n} - 2t_{m,n} + t_{m-1,n}}{\Delta x^2} + \cancel{O(\Delta x^2)}$$





(2) 由控制方程得到内节点 $(m,n)$ 的离散代数方程

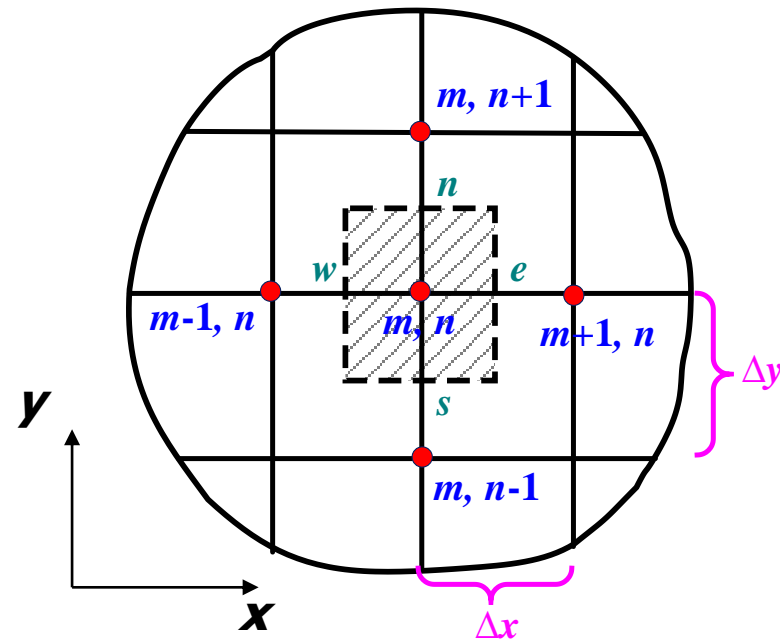
$$\frac{\partial^2 t}{\partial x^2} + \frac{\partial^2 t}{\partial y^2} = 0$$

### 二阶导数的差分表达式

$$x\text{方向} \quad \left. \frac{\partial^2 t}{\partial x^2} \right|_{m,n} = \frac{t_{m+1,n} - 2t_{m,n} + t_{m-1,n}}{\Delta x^2}$$

$$y\text{方向} \quad \left. \frac{\partial^2 t}{\partial y^2} \right|_{m,n} = \frac{t_{m,n+1} - 2t_{m,n} + t_{m,n-1}}{\Delta y^2}$$

$$\text{相加} \quad \left. \frac{\partial^2 t}{\partial x^2} \right|_{m,n} + \left. \frac{\partial^2 t}{\partial y^2} \right|_{m,n} = \frac{t_{m+1,n} - 2t_{m,n} + t_{m-1,n}}{\Delta x^2} + \frac{t_{m,n+1} - 2t_{m,n} + t_{m,n-1}}{\Delta y^2} = 0$$



$$\Delta x = \Delta y$$

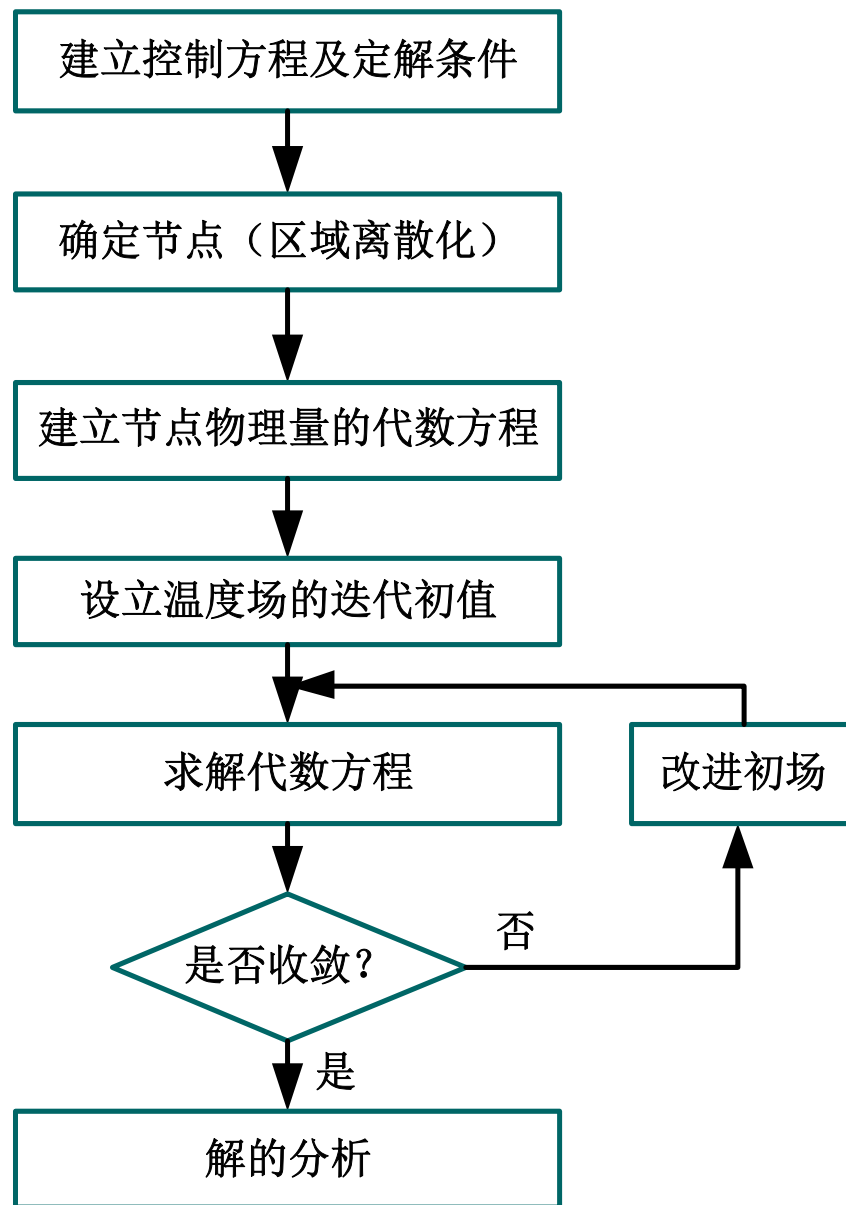


$$t_{m,n} = \frac{1}{4} (t_{m+1,n} + t_{m-1,n} + t_{m,n+1} + t_{m,n-1})$$

## 4.2 离散方程的建立方法

- (1) 控制容积热平衡法：着重于从物理的观点来分析
- (2) Taylor（泰勒）级数展开法：偏重于从数学的角度进行推导
- (3) 多项式拟合法
- (4) 控制容积积分法

# 导热问题数值求解基本步骤



4.1 数值求解的基本思想

4.2 离散方程的建立方法

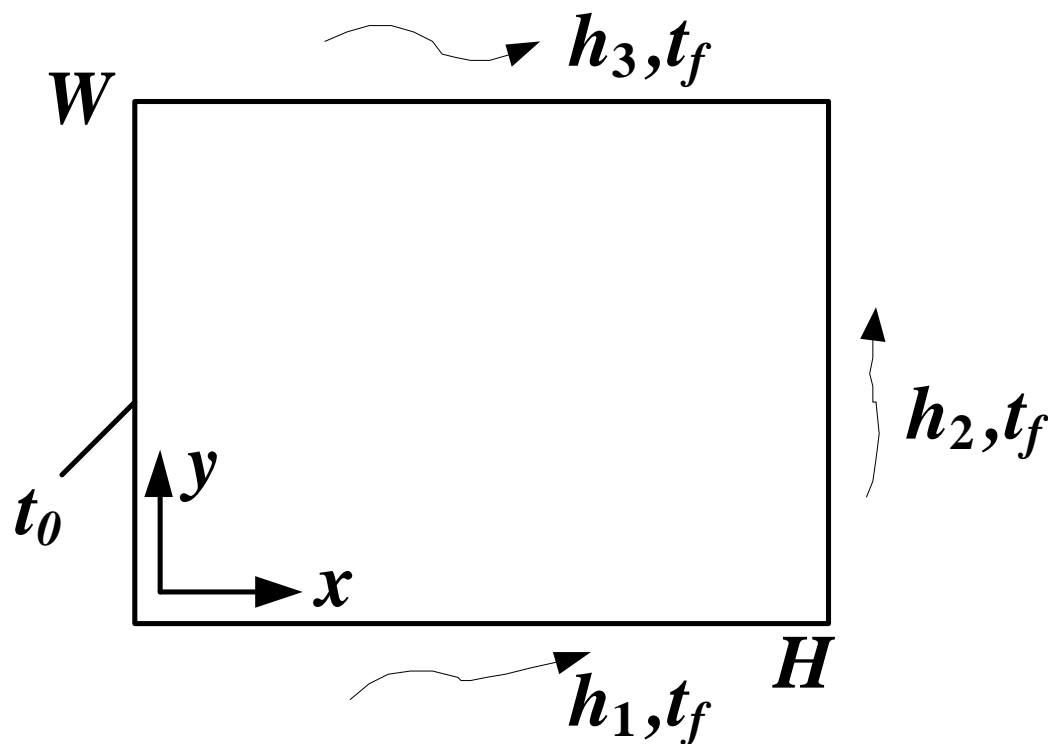
4.3 二维稳态导热问题的计算

4.4 代数方程组的解法

4.5 一维非稳态导热问题的计算

## 物理问题

二维矩形域内，稳态，无内热源，常物性的导热问题，第一、三类边界条件



# (1) 建立控制方程及定解条件

导热微分方程

$$\frac{\partial^2 t}{\partial x^2} + \frac{\partial^2 t}{\partial y^2} = 0$$

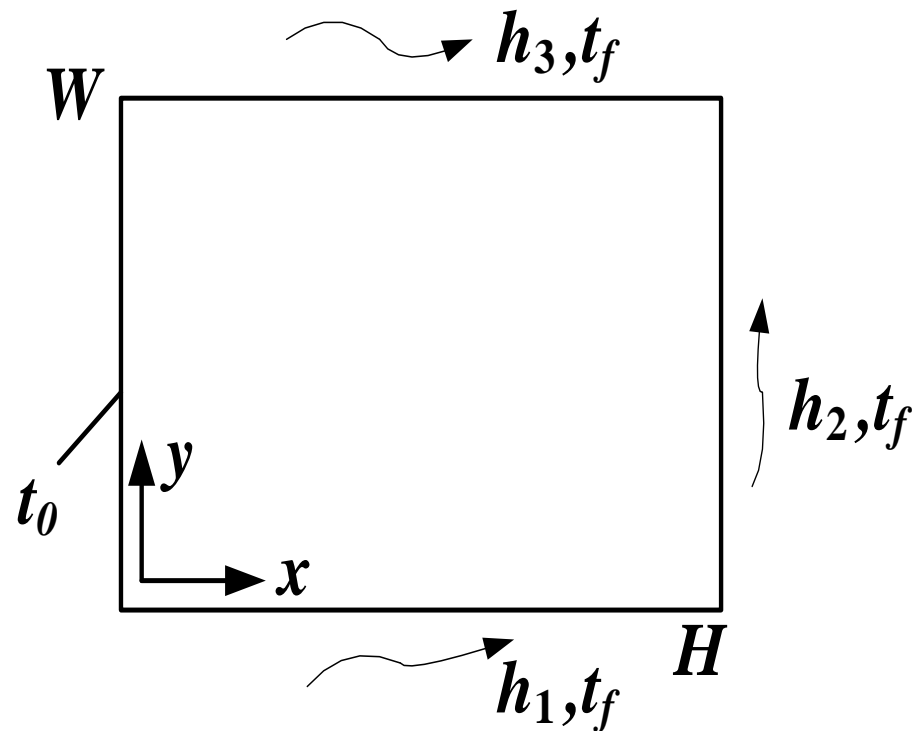
边界条件

$$x = 0, \quad t = t_0$$

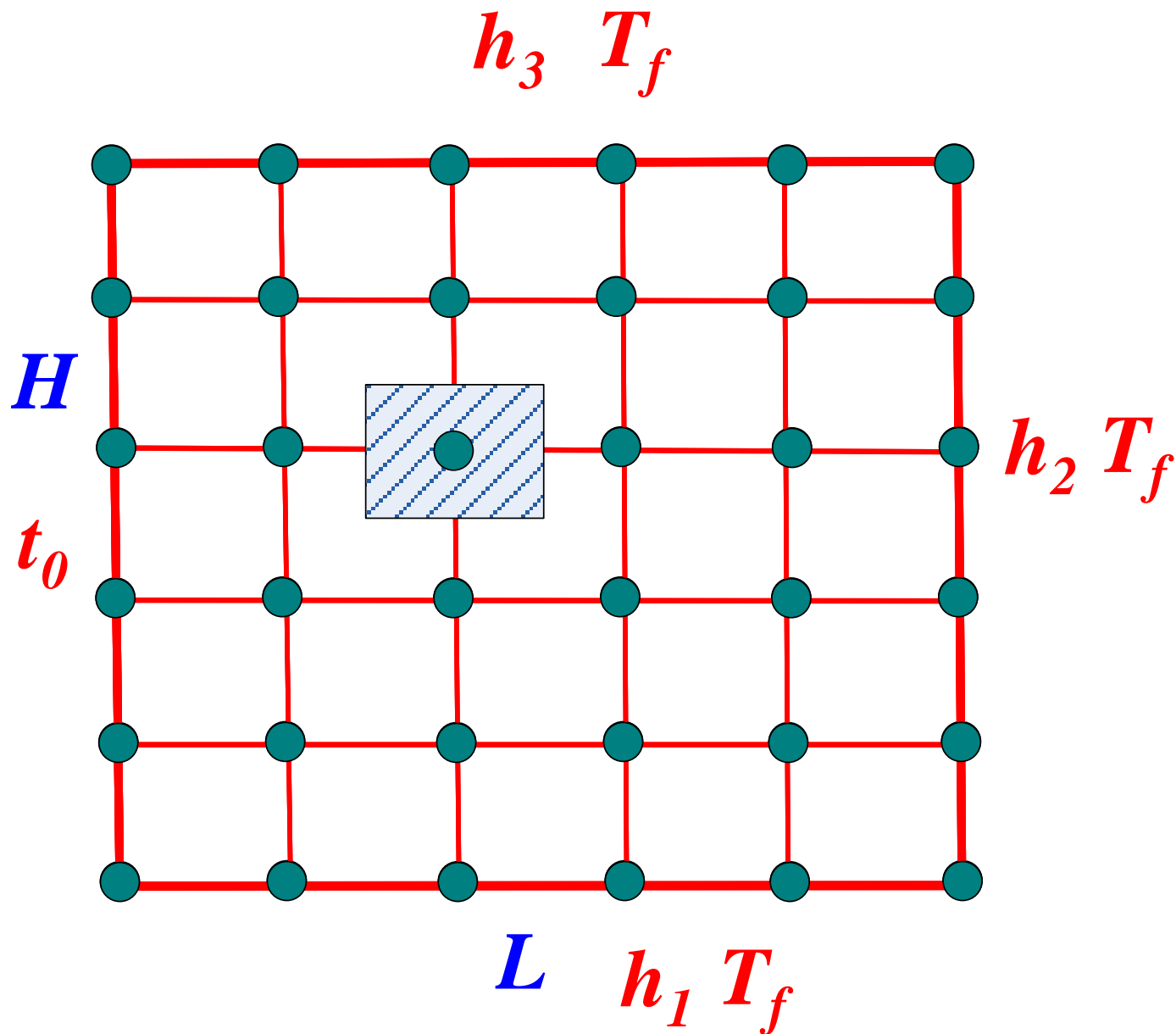
$$y = 0, \quad -\lambda \frac{\partial t}{\partial y} = h_1 (t - t_f)$$

$$x = H, \quad -\lambda \frac{\partial t}{\partial x} = h_2 (t - t_f)$$

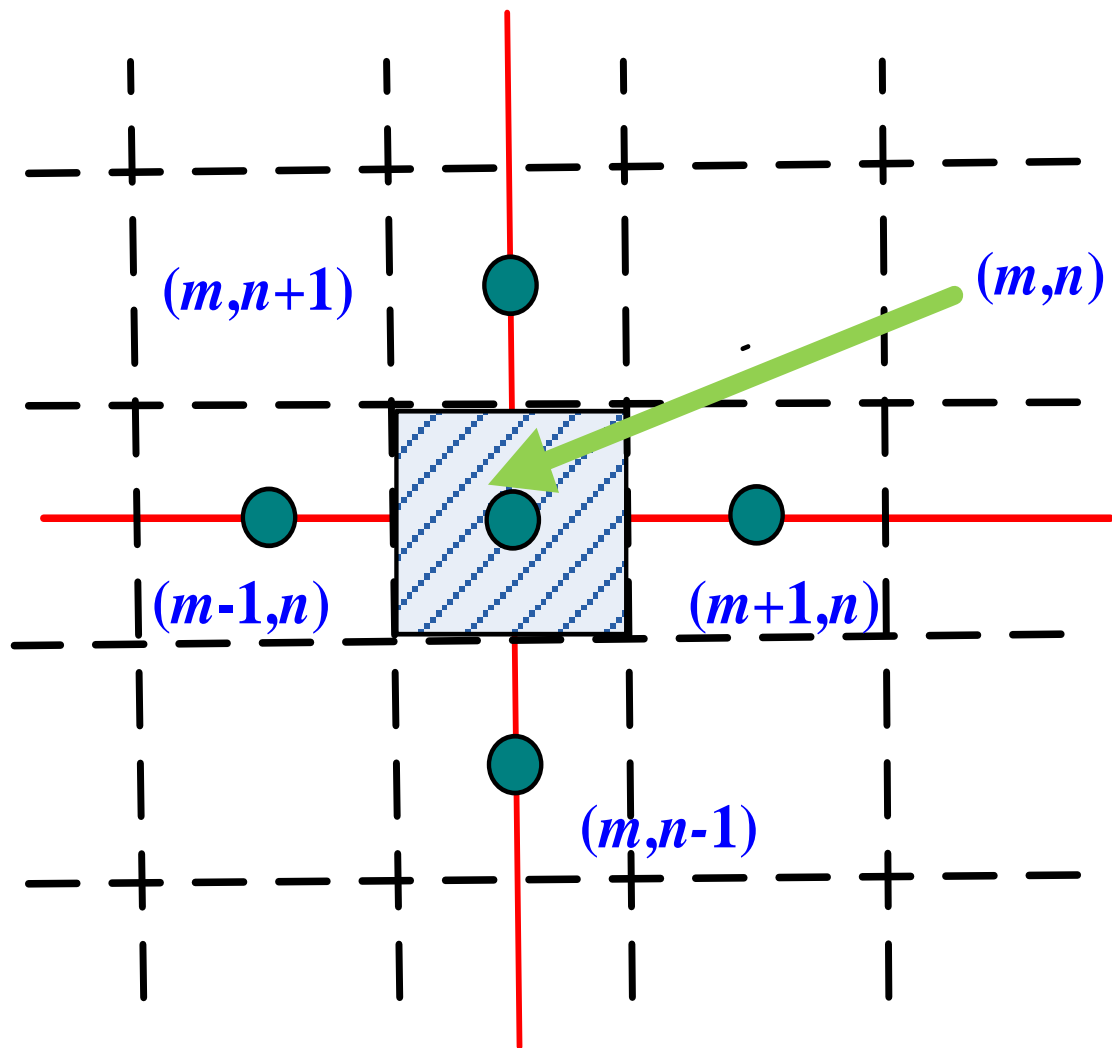
$$y = W, \quad -\lambda \frac{\partial t}{\partial y} = h_3 (t - t_f)$$



## (2) 区域离散化



### (3) 建立节点物理量的代数方程



$$\frac{\partial^2 t}{\partial x^2} + \frac{\partial^2 t}{\partial y^2} = 0$$



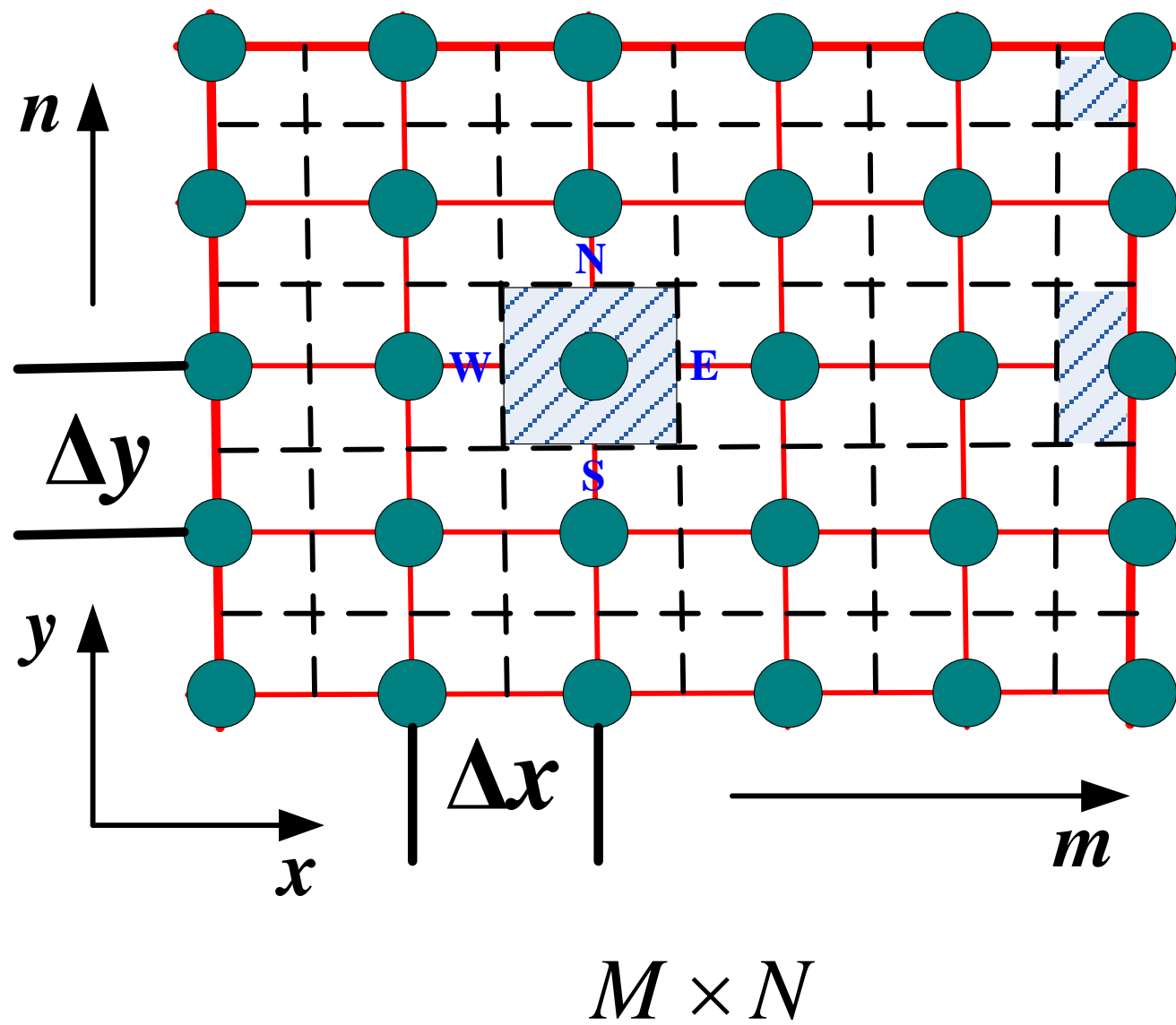
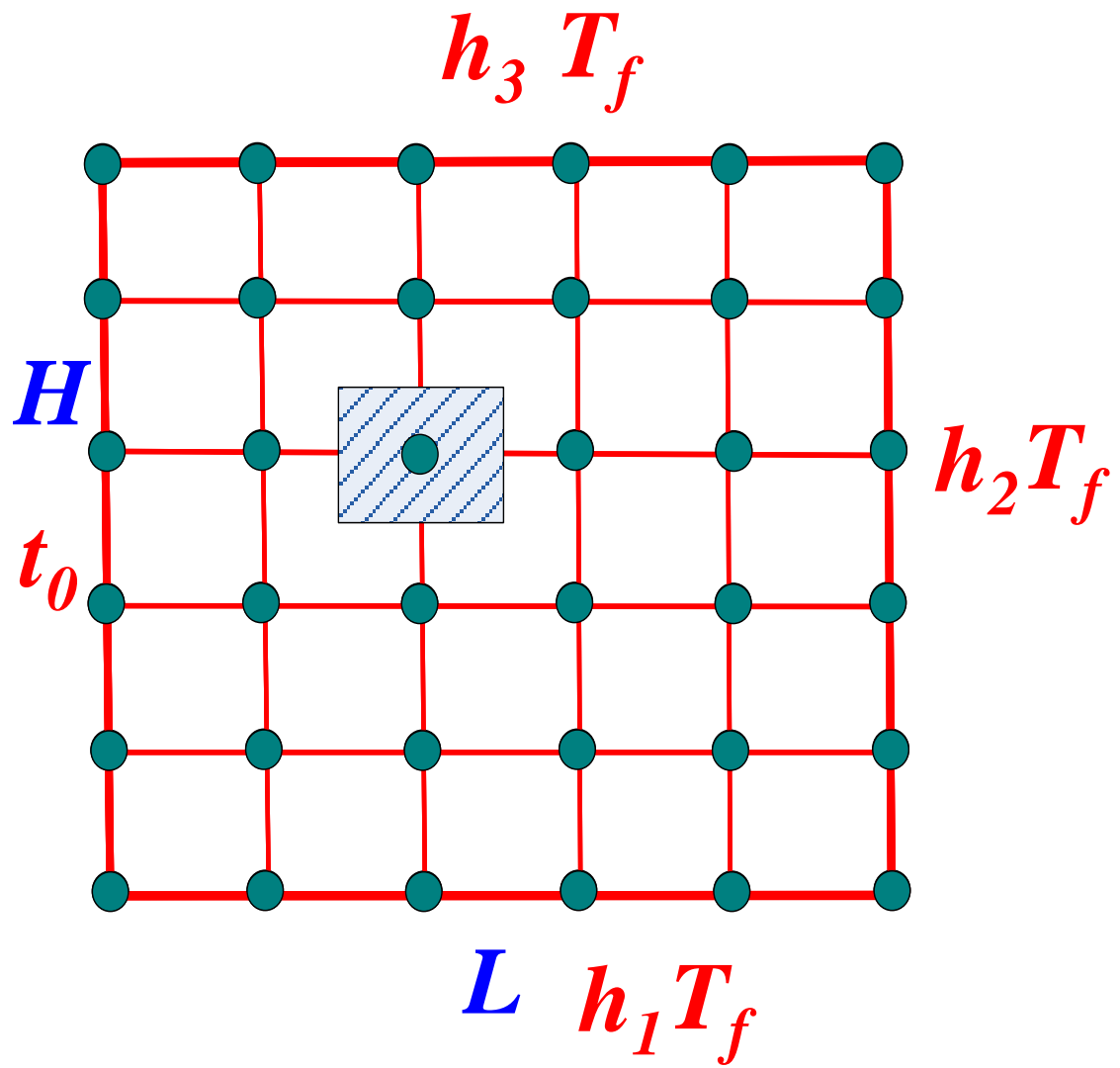
内节点

$$t_{m,n} = \frac{1}{4} (t_{m+1,n} + t_{m-1,n} + t_{m,n+1} + t_{m,n-1})$$

$$m = 2, \dots, M-1$$

$$n = 2, \dots, N-1$$





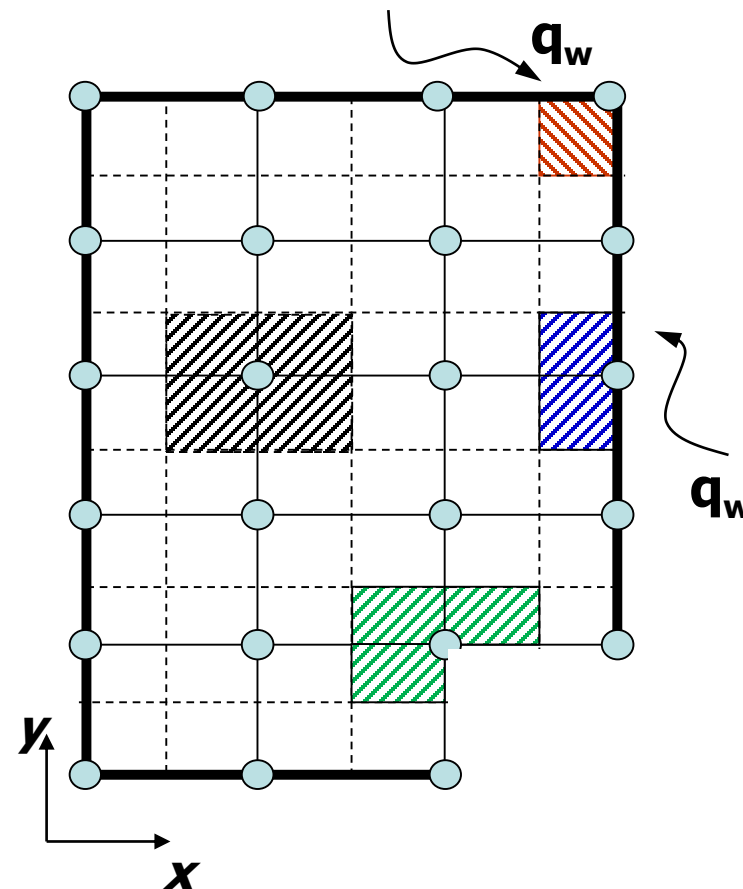
### (3) 建立边界节点物理量的代数方程

#### ➤ 第一类边界条件

边界节点温度给定，可将其以数值的形式加入内节点的离散方程中，组成封闭的代数方程组，可直接求解，因此处理较简单。

#### ➤ 第二类边界条件和第三类边界条件

用热平衡的方法，建立边界节点的离散方程，边界节点与内节点的离散方程一起组成封闭的代数方程组，才能求解。



# 边界节点的类型:

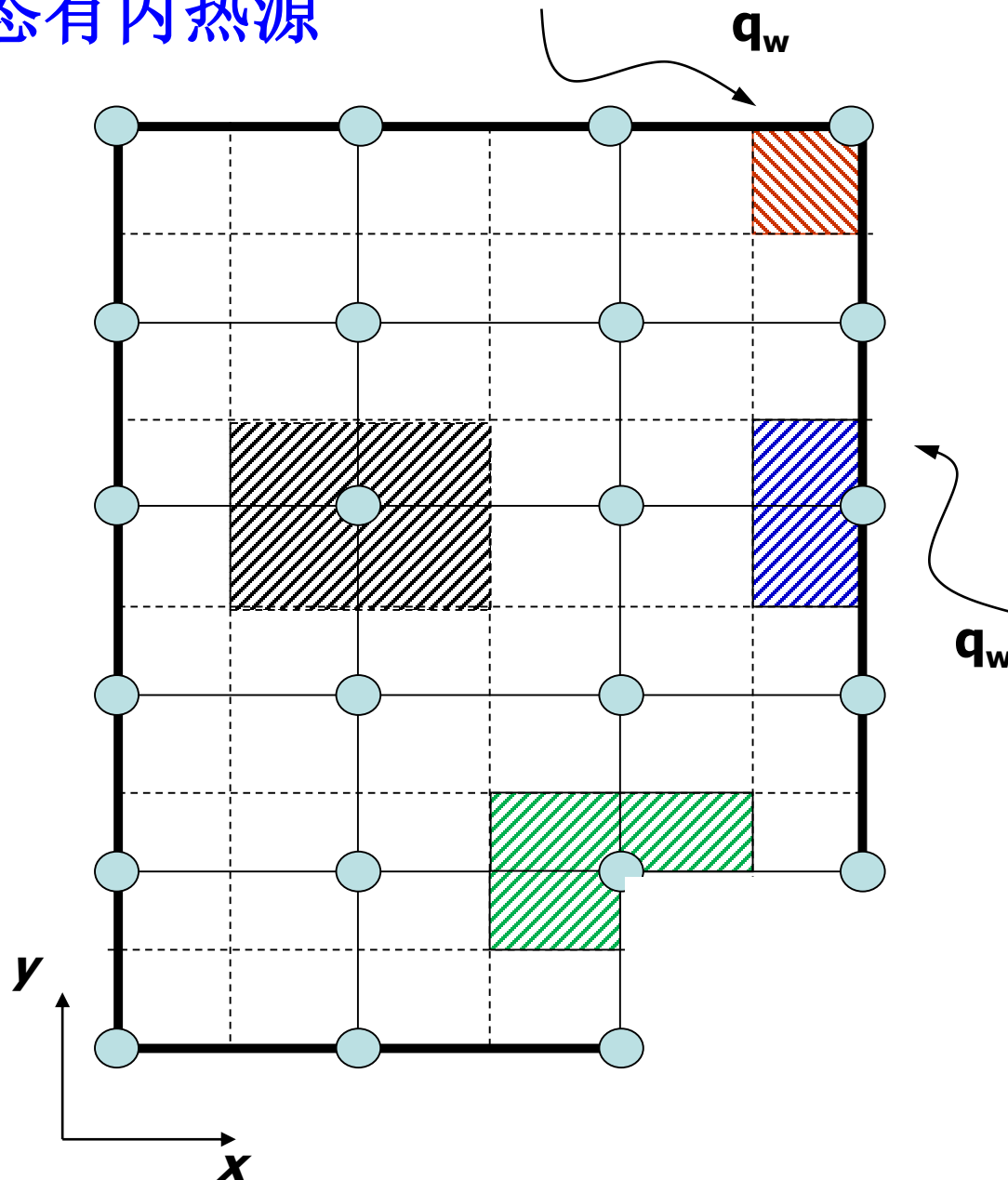
(1) 平直边界上的节点

(2) 外部角点

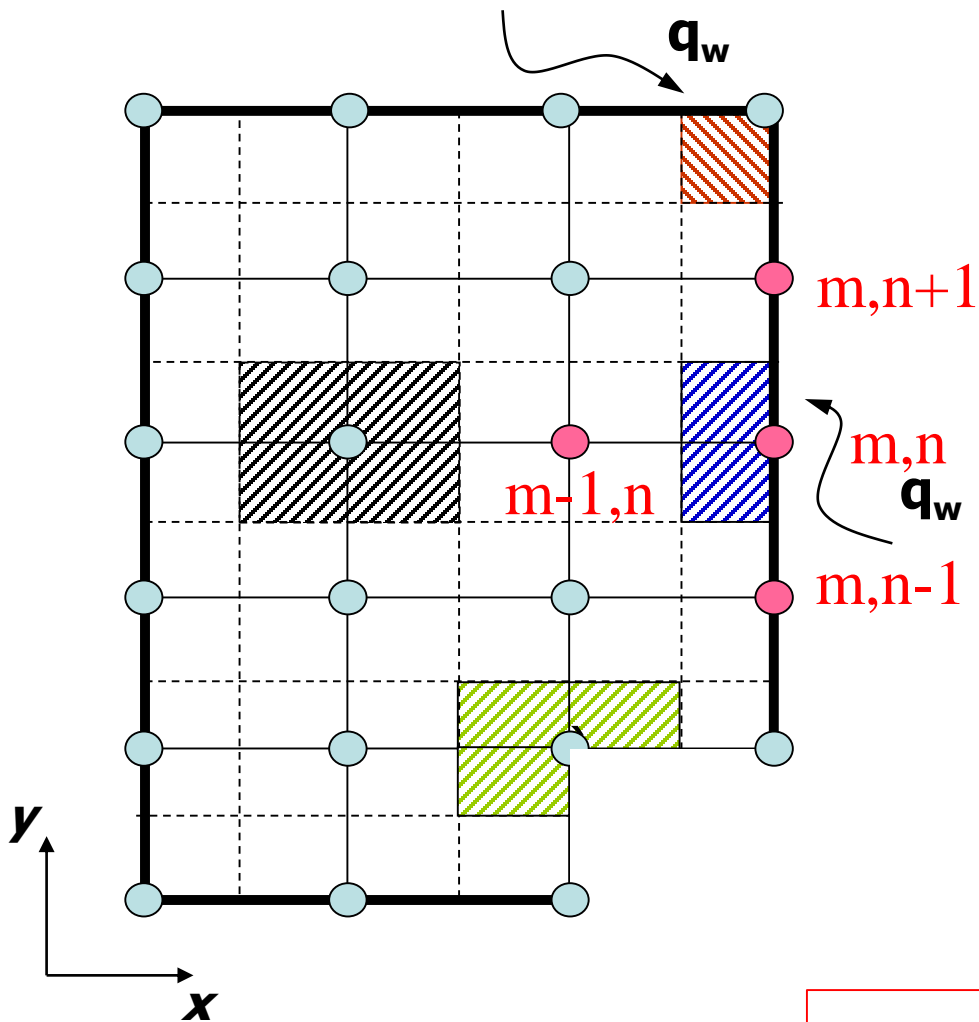
(3) 内部角点

注: 用 $q_w$ 表示边界上的热流密度  
用 $\Phi$ 表示内热源强度。

稳态有内热源



## (1) 平直边界上的节点



稳态有内热源

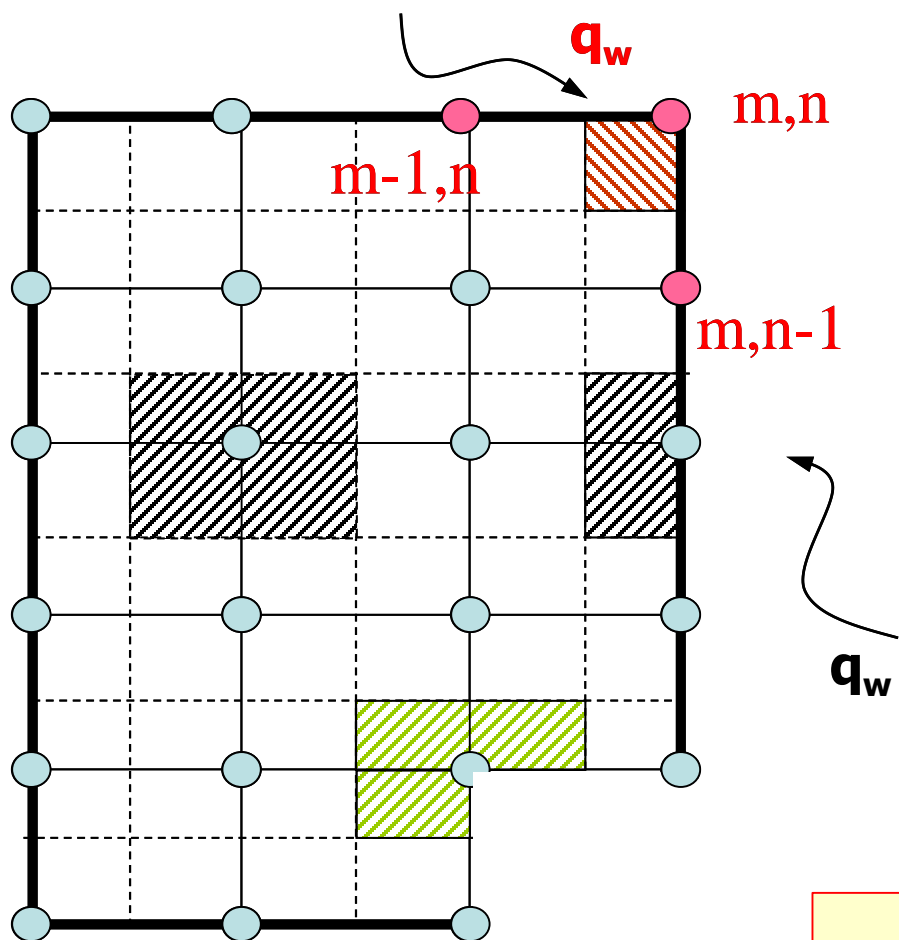
从所有方向流入控制体的总热量  
+ 控制体内热源生成热 = 0

$$\begin{aligned} & \lambda \Delta y \frac{t_{m-1,n} - t_{m,n}}{\Delta x} + \Delta y q_w \\ & + \lambda \frac{\Delta x}{2} \frac{t_{m,n+1} - t_{m,n}}{\Delta y} + \lambda \frac{\Delta x}{2} \frac{t_{m,n-1} - t_{m,n}}{\Delta y} \\ & + \dot{\Phi}_{m,n} \frac{\Delta x}{2} \Delta y = 0 \end{aligned}$$

$$\Delta x = \Delta y \Rightarrow$$

$$t_{m,n} = \frac{1}{4} \left( 2t_{m-1,n} + \frac{2\Delta x}{\lambda} q_w + t_{m,n+1} + t_{m,n-1} + \dot{\Phi}_{m,n} \frac{\Delta x^2}{\lambda} \right)$$

## (2) 外部角点



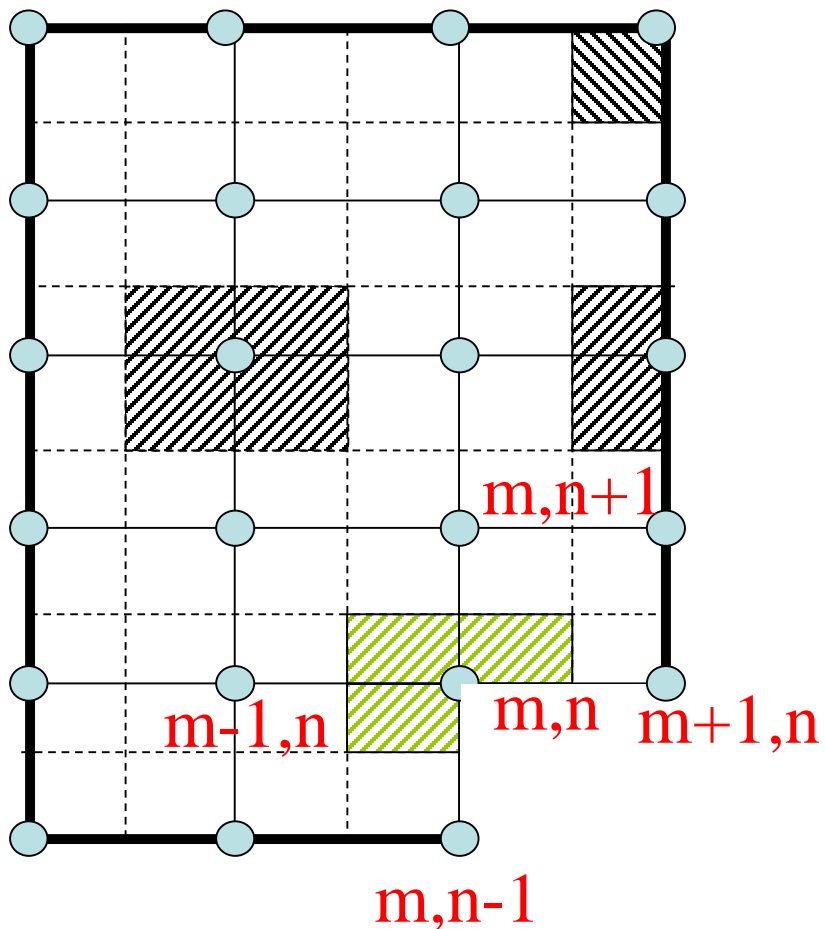
稳态有内热源

$$\lambda \frac{\Delta y}{2} \frac{t_{m-1,n} - t_{m,n}}{\Delta x} + \frac{\Delta y}{2} q_w + \frac{\Delta x}{2} q_w + \lambda \frac{\Delta x}{2} \frac{t_{m,n-1} - t_{m,n}}{\Delta y} + \dot{\Phi}_{m,n} \frac{\Delta x}{2} \cdot \frac{\Delta y}{2} = 0$$

$$\Delta x = \Delta y \Rightarrow$$

$$t_{m,n} = \frac{1}{2} (t_{m-1,n} + t_{m,n-1} + \frac{2\Delta x}{\lambda} q_w + \dot{\Phi}_{m,n} \frac{\Delta x^2}{2\lambda})$$

### 3) 内部角点



稳态有内热源

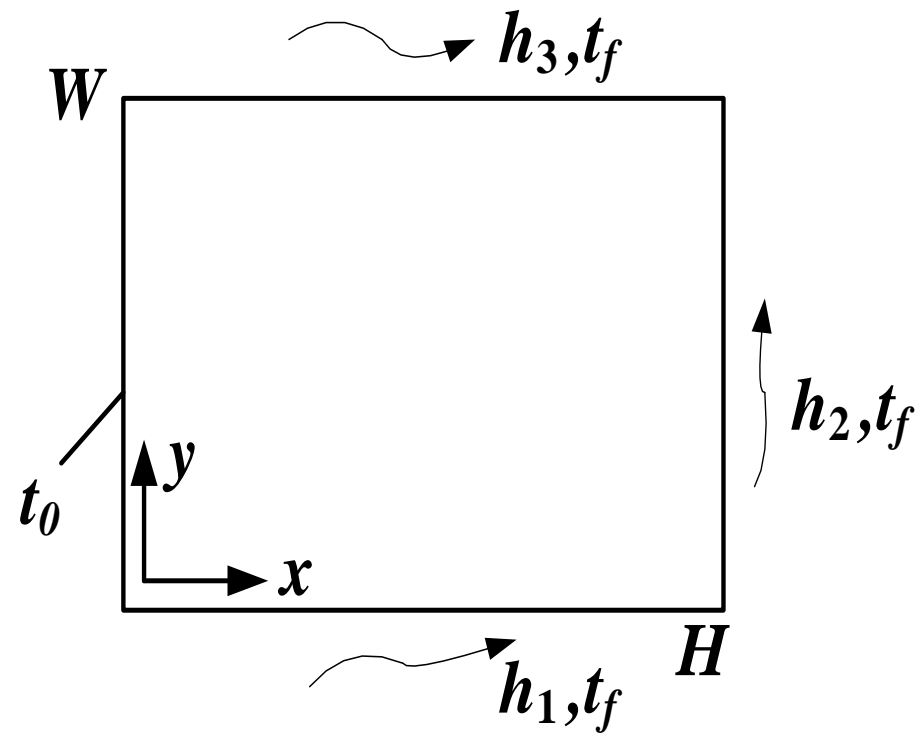
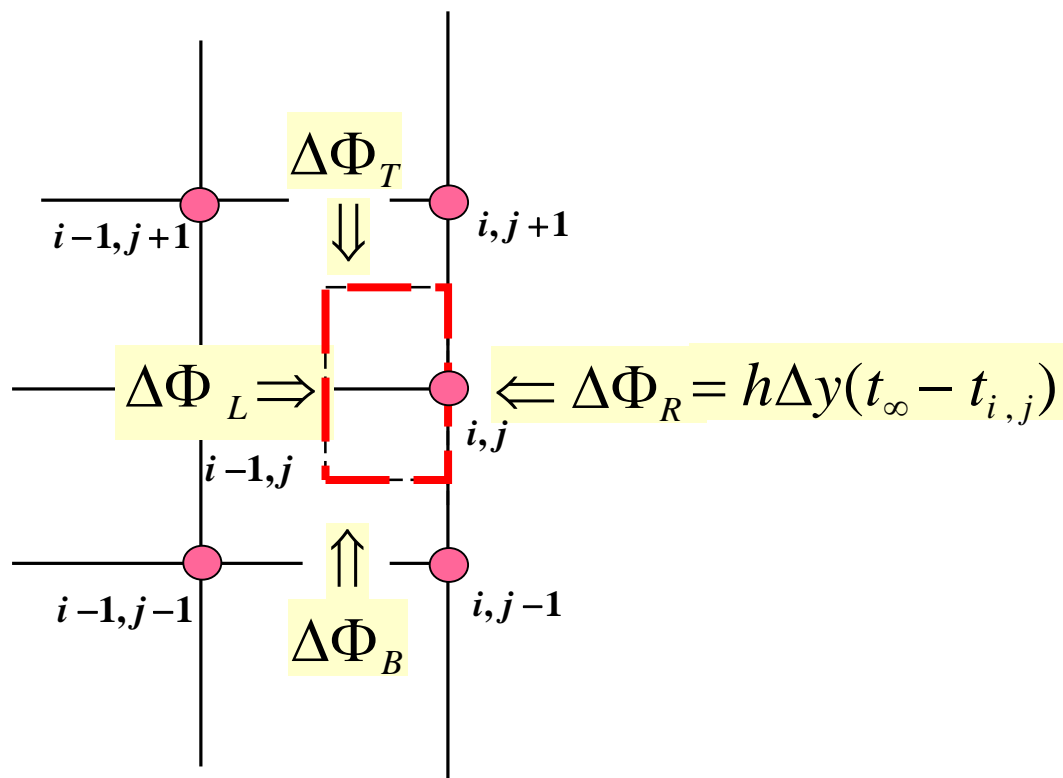
$$\lambda \Delta y \frac{t_{m-1,n} - t_{m,n}}{\Delta x} + \left( \lambda \frac{\Delta y}{2} \frac{t_{m+1,n} - t_{m,n}}{\Delta x} + \frac{\Delta y}{2} q_w \right) \\ + \lambda \Delta x \frac{t_{m,n+1} - t_{m,n}}{\Delta y} + \left( \lambda \frac{\Delta x}{2} \frac{t_{m,n-1} - t_{m,n}}{\Delta y} + \frac{\Delta x}{2} q_w \right) \\ + \dot{\Phi}_{m,n} \frac{3\Delta x \Delta y}{4} = 0$$

$$\Delta x = \Delta y \Rightarrow$$

$$t_{m,n} = \frac{1}{6} (2t_{m-1,n} + 2t_{m,n+1} + t_{m,n-1} + t_{m+1,n} \\ + \frac{3\Delta x^2}{2\lambda} \dot{\Phi}_{m,n} + \frac{2\Delta x}{\lambda} q_w)$$

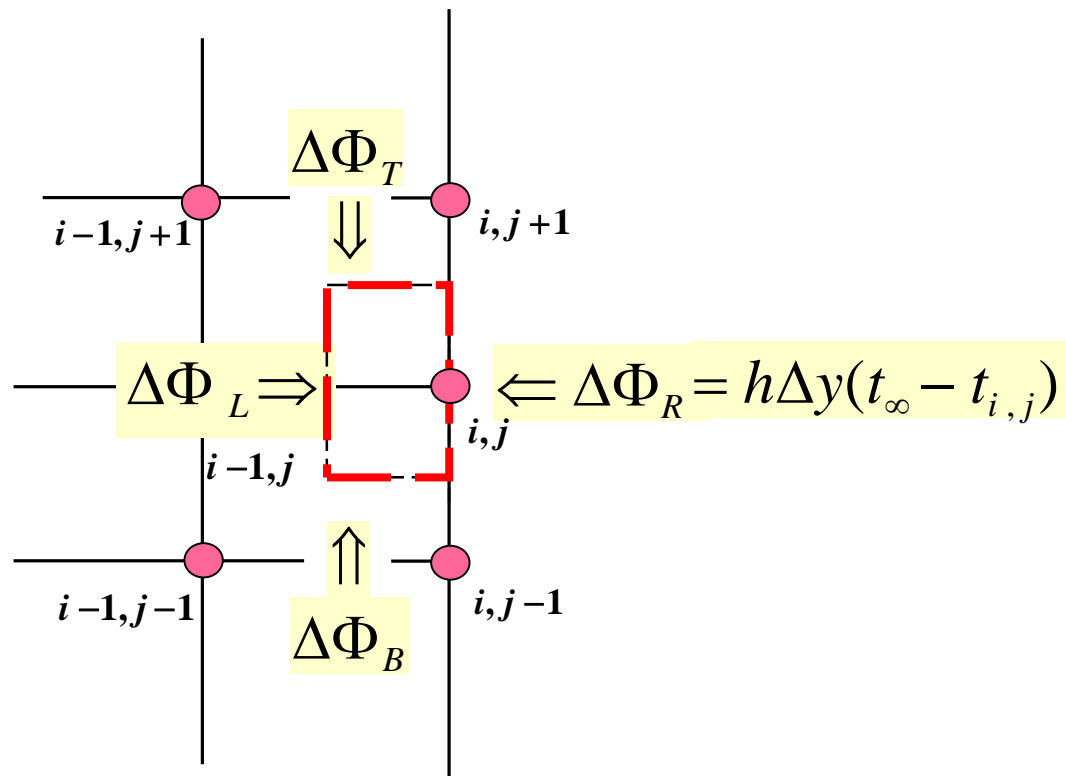
稳态无内热源条件

$$\Delta\Phi_L + \Delta\Phi_R + \Delta\Phi_B + \Delta\Phi_T = 0$$



假设  $\Delta x = \Delta y$   
试计算  $(i,j)$  点温度  $t_{i,j}$





$$\left\{ \begin{array}{l} \Delta\Phi_L = -\lambda\Delta y \left( \frac{t_{i,j} - t_{i-1,j}}{\Delta x} \right), \\ \Delta\Phi_R = -h\Delta y (t_{i,j} - t_\infty) \\ \Delta\Phi_B = -\lambda \frac{\Delta x}{2} \left( \frac{t_{i,j} - t_{i,j-1}}{\Delta y} \right), \\ \Delta\Phi_T = \lambda \frac{\Delta x}{2} \left( \frac{t_{i,j+1} - t_{i,j}}{\Delta y} \right) \end{array} \right.$$

$$-\lambda\Delta y \left( \frac{t_{i,j} - t_{i-1,j}}{\Delta x} \right) + h\Delta y (t_\infty - t_{i,j}) - \lambda \frac{\Delta x}{2} \left( \frac{t_{i,j} - t_{i,j-1}}{\Delta y} \right) + \lambda \frac{\Delta x}{2} \left( \frac{t_{i,j+1} - t_{i,j}}{\Delta y} \right) = 0$$

若  $\Delta x = \Delta y$

$$-\left(4 + \frac{2h\Delta x}{\lambda}\right)t_{i,j} + 2t_{i-1,j} + t_{i,j-1} + t_{i,j+1} + \frac{2h\Delta x}{\lambda}t_\infty = 0$$