

USEFUL EQUATIONS FOR THE ABIH EXAMINATIONS

This list of equations is offered as assistance in taking the ABIH examinations. No assurance is given that this list is complete or that the use of this list will assure the successful completion of any examination. The variables used are the same as found in the reference source for the equation. No attempt has been made to standardize variables.
 [Metric (SI) equations are in brackets]

VENTILATION

$$1 Q = VA \quad 2 V_1 A_1 = V_2 A_2 \quad 3 TP = VP + SP \quad 4 SP_1 + VP_1 = SP_2 + VP_2 + \sum losses_{1-2} \quad 5 SP_h = -\left((F_h + 1)VP_d \right)$$

$$6 V = 4005 \sqrt{\frac{VP}{df}} \quad 7 \left[V = 1.29 \sqrt{\frac{VP}{df}} \right] \quad 8 VP = \left(\frac{V}{4005} \right)^2 df \quad 9 \left[VP = \left(\frac{V}{1.29} \right)^2 df \right] \quad 10 hood entry loss = F_h x VP_d$$

$$11 C_e = \sqrt{\frac{VP}{|SP_h|}} \quad 12 VP_r = \left(\frac{Q_1}{Q_3} \right) VP_1 + \left(\frac{Q_2}{Q_3} \right) VP_2 \quad 13 Q = 4005(C_e) \sqrt{\frac{|SP_h|}{df}}(A) \quad 14 \left[Q = 1.29(C_e) \sqrt{\frac{|SP_h|}{df}}(A) \right]$$

$$15 Q = 4005C_e A \sqrt{|SP_h|} \quad 16 Q_{corr} = Q_{lower} \sqrt{\frac{SP_{gov}}{SP_{lower}}} \quad 17 Q' = \frac{Q}{m_i} \quad 18 t_2 - t_1 = -\frac{V_r}{Q'} \ln \left(\frac{C_{g2}}{C_{g1}} \right)$$

$$19 \ln \frac{(G - Q'C_{g2})}{(G - Q'C_{g1})} = -\frac{Q'(t_2 - t_1)}{V_r} \quad 20 Q = \frac{(403)(SG)(ER)(m_i)(10^6)}{(MW)(C_g)} \quad 21 \left[Q = \frac{(24)(SG)(ER)(m_i)(10^6)}{(MW)(C_g)} \right]$$

$$22 N_{changes} = \frac{60Q}{V_r} \quad 23 C_{g2} = \frac{G \left(1 - e^{-\left(\frac{Q' \Delta t}{V_r} \right)} \right)}{Q'} \quad 24 C_{g2} = C_{g1} e^{-\left(\frac{Q' \Delta t}{V_r} \right)} \quad 25 Q_2 = Q_1 \left(\frac{d_2}{d_1} \right)^3 \left(\frac{RPM_2}{RPM_1} \right)$$

$$26 P_2 = P_1 \left(\frac{d_2}{d_1} \right)^2 \left(\frac{RPM_2}{RPM_1} \right)^2 \quad 27 PWR_2 = PWR_1 \left(\frac{d_2}{d_1} \right)^5 \left(\frac{RPM_2}{RPM_1} \right)^3 \quad 28 FSP = SP_{out} - SP_{in} - VP_{in} \quad 29 FTP = TP_{out} - TP_{in}$$

NOISE

$$1 SPL \text{ or } L_p = 20 \log \left(\frac{P}{P_0} \right) \quad 2 L_I = 10 \log \left(\frac{I}{I_0} \right) \quad 3 SPL_2 = SPL_1 + 20 \log \left(\frac{d_1}{d_2} \right) \quad 4 L_w = 10 \log \left(\frac{W}{W_0} \right)$$

$$5 W_0 = 10^{-12} watts \quad 6 L_{eq} = 10 \log \left(\frac{1}{T} \sum_{i=1}^N \left(10^{\frac{L_i}{10}} t_i \right) \right) \quad 7 L_{PT} = 10 \log \left(\sum_{i=1}^N 10^{\frac{L_{Pi}}{10}} \right) \quad 8 TL = 10 \log \left(\frac{1}{\tau} \right)$$

$$9 L_p = L_w - 20 \log r - 0.5 + DI + CF \quad 10 [L_p = L_w - 20 \log r - 11 + DI + CF] \quad 11 DI = 10 \log Q$$

$$12 \%D = 100 \left(\frac{C_1}{T_1} + \frac{C_2}{T_2} + \dots + \frac{C_i}{T_i} \right) \quad 13 T_p = \frac{T_c}{2^{(L_{AS}-L_c)/ER}} \quad 14 TWA_{eq} = 10 \log \left(\frac{\%D}{100} \right) + 85 dBA$$

$$15 TWA = 16.61 \log \left(\frac{\%D}{100} \right) + 90 dBA \quad 16 f = \frac{(N)(RPM)}{60} \quad 17 f = \frac{c}{\lambda} \quad 18 f_2 = 2f_1 \quad 19 f_c = \sqrt{f_1 f_2} \quad 20 f_2 = \sqrt[3]{2} f_1$$

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GENERAL SCIENCES, STATISTICS, STANDARDS

$$1 \ ppm = \frac{V_{contam}}{V_{air}} \times 10^6 \quad 2 \ ppm = \frac{P_v}{P_{atm}} \times 10^6 \quad 3 \ ppm = \frac{mg/m^3 \times 24.45}{m.w.} \quad 4 \ \frac{P_1 V_1}{nRT_1} = \frac{P_2 V_2}{nRT_2} \quad 5 \ V_{TS} = \frac{gd_p^2 (\rho_p - \rho_a)}{18\eta}$$

$$6 \ R_e = \frac{\rho dv}{\eta} \quad 7 \ log \frac{I_o}{I} = abc \quad 8 \ pH = -log_{10}[H^+] \quad 9 \ K_a = \frac{[H^+]x[A^-]}{[HA]} \quad 10 \ K_b = \frac{[BH^+]x[OH^-]}{[B]}$$

$$11 \ P_{total} = X_1 P_1 + X_2 P_2 + \dots + X_i P_i \quad 12 \ vapor/hazard \ ratio = \frac{sat. \ concentration}{exposure \ guideline} \quad 13 \ TLV_{mix} = \frac{C_1}{TLV_1} + \frac{C_2}{TLV_2} + \dots + \frac{C_n}{TLV_n}$$

$$14 \ TLV_{mix} = \frac{1}{\frac{F_1}{TLV_1} + \frac{F_2}{TLV_2} + \dots + \frac{F_n}{TLV_n}} \quad 15 \ RF = \frac{8}{h} \times \frac{24-h}{16} \quad 16 \ RF = \frac{40}{h_w} \times \frac{168-h_w}{128} \quad 17 \ C_{asb} = \frac{(C_s - C_b)A_c}{1000A_fV_s}$$

$$18 \ C_{asb} = \frac{EA_c}{1000V_s} \quad 19 \ E_{fiber \ density} = \frac{\frac{f}{N_f} - \frac{B}{N_b}}{A_f} \quad 20 \ d = \frac{0.61\lambda}{\eta \sin \alpha} \quad 21 \ SD = \sqrt{\frac{\sum(\bar{x} - x_i)^2}{n-1}} \quad 22 \ GM = \sqrt[n]{(x_1)(x_2)\dots(x_n)}$$

$$23 \ GM = 10^{\frac{\sum(\log x)}{n}} \quad 24 \ GSD = \frac{84.13\% \ tile \ value}{50\% \ tile \ value} \quad 25 \ GSD = \frac{50\% \ tile \ value}{15.87\% \ tile \ value} \quad 26 \ SAE = 1.645CV_{total} \quad 27 \ CV = \frac{SD}{\bar{X}}$$

$$28 \ E_c = \sqrt{E_1^2 + E_2^2 + \dots + E_n^2} \quad 29 \ t = \frac{\bar{x}_1 - \bar{x}_2}{SD_{pooled} \sqrt{\frac{1}{n_1} + \frac{1}{n_2}}} \quad 30 \ SD_{pooled} = \sqrt{\frac{(n_1 - 1)SD_1^2 + (n_2 - 1)SD_2^2}{n_1 + n_2 - 2}}$$

$$31 \ LCL = \frac{C_A}{PEL} - \frac{SAE \sqrt{T_1^2 C_1^2 + T_2^2 C_2^2 + \dots + T_n^2 C_n^2}}{PEL(T_1 + T_2 + \dots + T_n)} \quad 32 \ RWL = LCxHMxVMxDMxAMxFMxCM \quad 33 \ LI = \frac{L}{RWL}$$

$$34 \ 90\% Conf \ Interval = \bar{X} \pm 1.645 \frac{SD}{\sqrt{n}} \quad 35 \ 95\% Conf \ Limit = \bar{X} + 1.645 \frac{SD}{\sqrt{n}}$$

HEAT STRESS

$$1 \ WBGT = 0.7t_{nwb} + 0.2t_g + 0.1t_{db} \quad 2 \ WBGT = 0.7t_{nwb} + 0.3t_g \quad 3 \ \Delta S = (M - W) \pm C \pm R - E$$

$$4 \ Q_s = \frac{H_s}{1.08x\Delta T} \quad 5 \left[Q_s = \frac{H_s}{20x\Delta T} \right]$$

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RADIATION

$$1 \ I_2 = I_1 \left(\frac{d_1}{d_2} \right)^2 \quad 2 \ Rem = (RAD)(QF) \quad 3 \ D = \frac{\Gamma A}{d^2} \quad 4 \ A = A_i(0.5)^{\frac{t}{T_{1/2}}} \quad 5 \ A_i = \frac{0.693}{T_{1/2}} N_i \quad 6 \ A = A_i e^{-\frac{0.693t}{T_{1/2}}}$$

$$7 \ I = (1/2)^A I_0 \quad 8 \ I = (1/10)^B I_0 \quad 9 \ I_2 = \frac{I_1}{2^{HVL}} \quad 10 \ I_2 = \frac{I_1}{10^{TVL}} \quad 11 \ X = 3.32 \log \left(\frac{I_1}{I_2} \right) (HVL) \quad 12 \ I = I_0 B e^{-ux}$$

$$13 \ \frac{1}{T_{1/2eff}} = \frac{1}{T_{1/2rad}} + \frac{1}{T_{1/2bio}} \quad 14 \ T_{1/2eff} = \frac{(T_{1/2rad})(T_{1/2bio})}{T_{1/2rad} + T_{1/2bio}} \quad 15 \ S = \frac{E^2}{3770} \quad 16 \ S = 37.7 H^2 \quad 17 \ S = \frac{4P}{A}$$

$$18 \ r = \left(\frac{PG}{4\pi EL} \right)^{1/2} \quad 19 \ r_{NHZ} = \frac{1}{\emptyset} \left(\frac{4\Phi}{\pi EL} - a^2 \right)^{1/2} \quad 20 \ r_{NHZ} = \frac{f_0}{b_0} \left(\frac{4\Phi}{\pi EL} \right)^{1/2} \quad 21 \ r_{NHZ} = \left(\frac{\rho \Phi \cos \theta}{\pi EL} \right)^{1/2}$$

$$22 \ D_s = \frac{1}{\emptyset} \left(\frac{4\Phi}{\pi TL} - a^2 \right)^{1/2} \quad 23 \ spatial \ ave = \left(\frac{\sum_{i=1}^N FS_i^2}{N} \right)^{1/2} \quad 24 \ t = \frac{0.003 J/cm^2}{E_{eff}} \quad 25 \ t = \frac{EL}{ML} \times 0.1h \quad 26 \ O.D. = \log \frac{I_0}{I}$$

$$27 \ D_L = \sqrt{a^2 + \emptyset^2 r^2} \quad 28 \ G = 10^{g/10}$$

CONSTANTS AND CONVERSIONS

$${}^\circ F = 9/5({}^\circ C) + 32 \quad {}^\circ R = {}^\circ F + 460 \quad K = {}^\circ C + 273.15 \quad \text{molar volume at } 25{}^\circ C, 1 \text{ atm} = 24.45 \text{ L} \quad 1 \text{ ft}^3 = 28.32 \text{ L}$$

$$1 \text{ ft}^3 = 7.481 \text{ U.S. gal} \quad 1 \text{ L} = 1.0566 \text{ qt} \quad 1 \text{ inch} = 2.54 \text{ cm} \quad 1 \text{ lb} = 453.6 \text{ grams} \quad 1 \text{ gram} = 15.43 \text{ grains}$$

$$1 \text{ atm} = 14.7 \text{ psi} = 760 \text{ mm Hg} = 29.92 \text{ in Hg} = 33.93 \text{ ft water} = 1013.25 \text{ mbar} = 101,325 \text{ pascals}$$

$$1 \text{ Curie} = 3.7 \times 10^{10} \text{ disint/sec (Becquerel)} = 2.2 \times 10^{12} \text{ dpm} \quad 1 \text{ Gray} = 100 \text{ Rad} \quad 1 \text{ Sievert} = 100 \text{ Rem}$$

$$1 \text{ Tesla} = 10,000 \text{ Gauss} \quad 1 \text{ BTU} = 1054.8 \text{ joules} = 0.293 \text{ watt hr} \quad 1 \text{ cal} = 4.184 \text{ joules}$$

$$\text{speed of sound in air at } 68{}^\circ F (20{}^\circ C) = 1130 \text{ fps (344 m/s)} \quad \text{speed of light} = 3 \times 10^8 \text{ m/s}$$

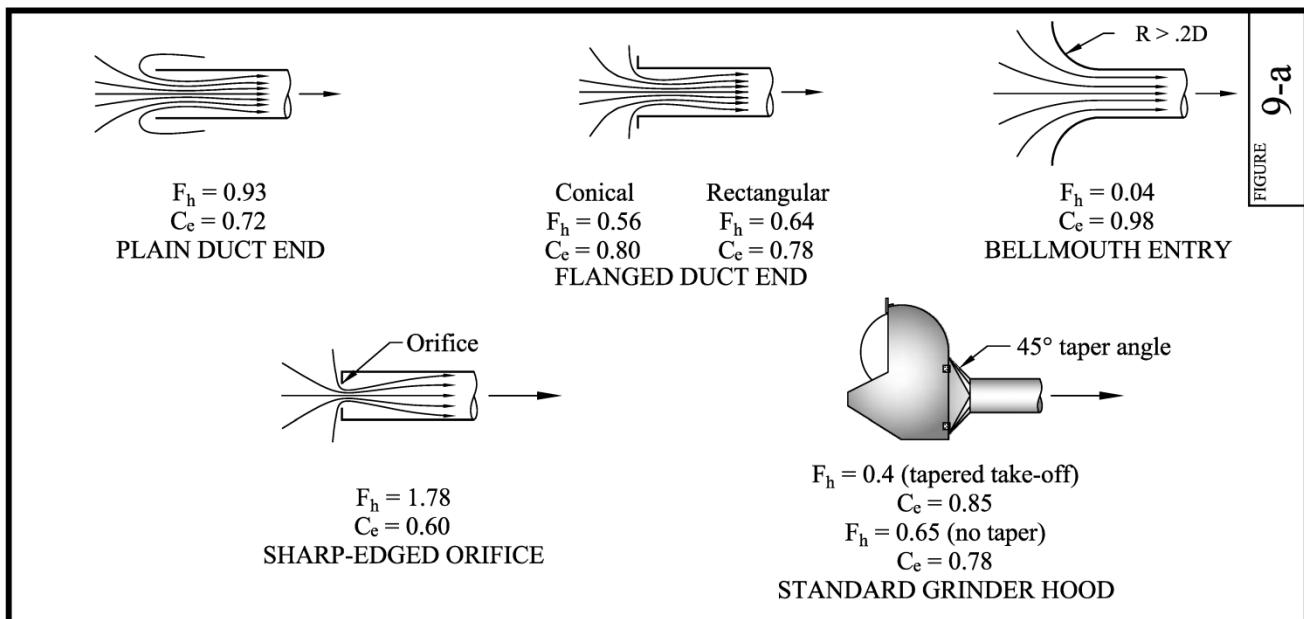
$$\text{Planck's constant} = 6.626 \times 10^{-27} \text{ erg sec} \quad \text{Avogadro's number} = 6.024 \times 10^{23}$$

$$\text{gas constant, R} = 8.314 \text{ J/mole K} = 0.082 \text{ L atm/mole K} \quad \text{density of air} = 1.29 \text{ g/L at atm, } 0{}^\circ C$$

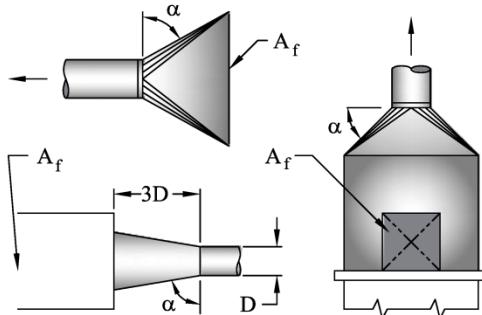
$$g = 981 \text{ cm/sec}^2 = 32 \text{ ft/sec}^2 \quad A_c = 385 \text{ mm}^2 \text{ for 25 mm filter} \quad Af = 0.00785 \text{ mm}^2$$

TABLE 6-3. Summary of Hood Airflow Equations

HOOD TYPE	DESCRIPTION	ASPECT RATIO, H/L	AIRFLOW
	Slot	0.2 or less	1 $Q = 3.7 LV_x X$
	Flanged slot	0.2 or less	2 $Q = 2.6 LV_x X$
	Plain opening	0.2 or greater and round	3 $Q = V_x(10X^2 + A_f)$ $A_f = WH$
	Flanged opening $W_f \geq \sqrt{A_f}$	0.2 or greater and round	4 $Q = 0.75V_x(10X^2 + A_f)$ $A_f = WH$
	Booth	To suit work	5 $Q = VA = V_f WH$
	Canopy	To suit work	6 $Q = 1.4 PVX$ P = Perimeter of work or tank X = Height above work
	Plain multiple slot opening (2) or more slots	0.2 or greater	7 $Q = V_x(10X^2 + A_s)$ $A_s = HL$
	Flanged multiple slot opening (2) or more slots	0.2 or greater	8 $Q = 0.75V_x(10X^2 + A_s)$ $A_s = HL$



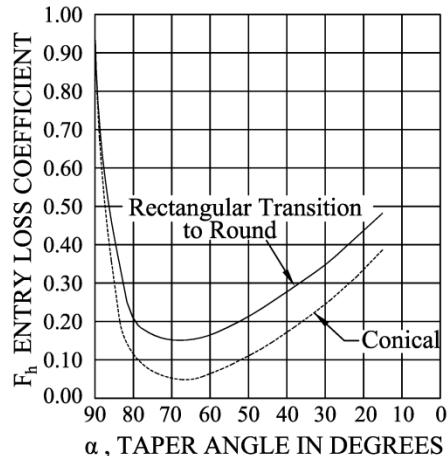
TAPERED HOODS
Flanged or unflanged; conical, square or rectangular. α is the major angle on rectangular hoods.



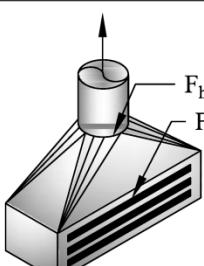
Face area (A_f) at least 2 times the duct area.

F_h Entry Loss Coefficient	α Conical Rectangular	
	Conical	Rectangular
90	0.93	0.93
80	0.11	0.19
70	0.06	0.13
60	0.08	0.16
50	0.12	0.21
45	0.15	0.25
40	0.18	0.28
30	0.25	0.35
20	0.34	0.43
10	0.44	0.52
0	0.56	0.64

Note: 0° values represent round ducts butted into back of booth or hood without a rectangular to round transition.



COMPOUND HOODS
A compound hood, such as the slot/plenum shown to the right, would have 2 losses, one through the slot and the other through the transition into the duct.
The slot entry loss coefficient, F_s , would have a value typically in the range of 1.00 to 1.78 (see Chapters 6 and 13).
The duct entry loss coefficient is given by the above data for tapered hoods.



$$\text{Hood Entry Loss} = F_s V P_s + F_h V P_d$$

(See Chapter 6)

MISCELLANEOUS VALUES

HOOD	ENTRY LOSS COEFFICIENT F_h	HOOD FLOW COEFFICIENT C_e
Abrasive blast chamber	1.00	0.50
Abrasive blast elevator	2.30	0.31
Abrasive separator	2.30	0.31
Elevators (enclosures)	1.00	0.50
Flanged pipe plus close elbow	0.80	0.56
Plain pipe plus close elbow	1.60	0.38



TITLE

HOOD ENTRY LOSS FACTORS

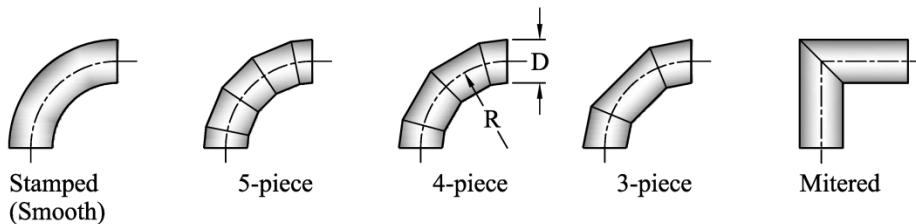
FIGURE

9-a

DATE

1-16

CHECK CODES, REGULATIONS, AND LAWS (LOCAL, STATE, AND NATIONAL)
TO ENSURE THAT DESIGN IS COMPLIANT.

FIGURE
9-e

	R / D				
	0.75	1.00	1.50	2.00	2.50
Stamped	0.33	0.22	0.15	0.13	0.12
5-piece	0.46	0.33	0.24	0.19	0.17*
4-piece	0.50	0.37	0.27	0.24	0.23*
3-piece	0.54	0.42	0.34	0.33	0.33*

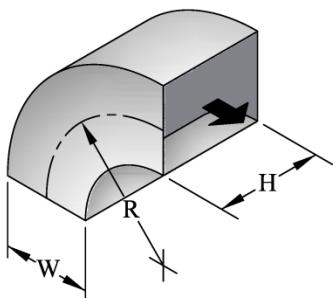
* extrapolated from published data

OTHER ELBOW LOSS FACTORS

Mitered, no vanes	1.2
Mitered, turning vanes	0.6
Flatback (R/D = 2.5)	0.05 (see Chapter 5, Figure 5-18)

NOTE: Loss factors are assumed to be for elbows of "zero length." Friction losses should be included to the intersection of centerlines.

ROUND ELBOW LOSS FACTORS



R / W	Aspect Ratio, H/W					
	0.25	0.5	1.0	2.0	3.0	4.0
0.0 (Mitered)	1.50	1.32	1.15	1.04	0.92	0.86
0.5	1.36	1.21	1.05	0.95	0.84	0.79
1.0	0.45	0.28	0.21	0.21	0.20	0.19
1.5	0.28	0.18	0.13	0.13	0.12	0.12
2.0	0.24	0.15	0.11	0.11	0.10	0.10
3.0	0.24	0.15	0.11	0.11	0.10	0.10

SQUARE & RECTANGULAR ELBOW LOSS FACTORS



TITLE

DUCT DESIGN DATA ELBOW LOSS FACTORS

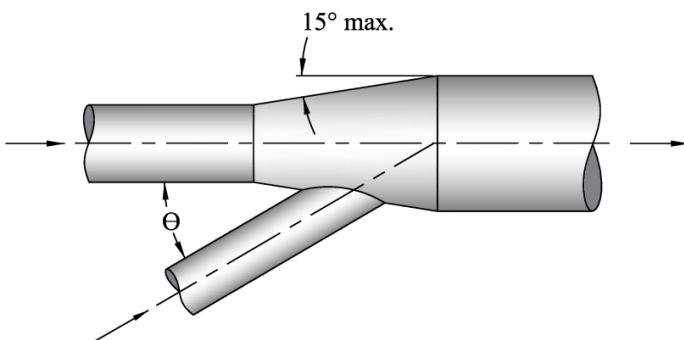
FIGURE

9-e

DATE

1-16

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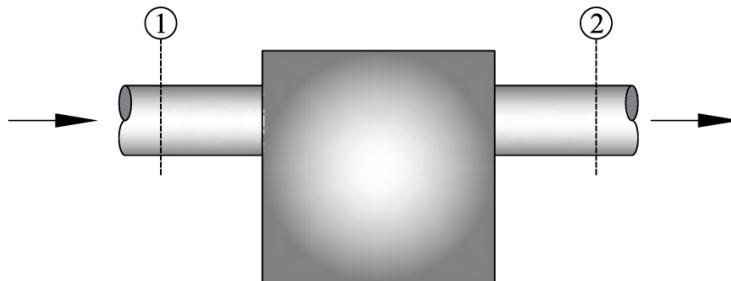
FIGURE
9-f

Note: Branch entry loss assumed to occur in branch and is so calculated.

Do not include a regain calculation for branch entry enlargements.

Angle Θ Degrees	Loss Factor
10	0.06
15	0.09
20	0.12
25	0.15
30	0.18
35	0.21
40	0.25
45	0.28
50	0.32
60	0.44
90	1.00

BRANCH ENTRY LOSS FACTORS



$$SP_2 - SP_1 = 1.5 VP_2$$

$$F_h = 1.5$$

$$C_e = 0.8$$

TRAP OR SETTLING CHAMBER



TITLE

BRANCH ENTRY LOSS FACTORS AND LOSSES IN SETTLING CHAMBERS

FIGURE

9-f

DATE

1-16

CHECK CODES, REGULATIONS, AND LAWS (LOCAL, STATE, AND NATIONAL)
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