

# Assignment\_1

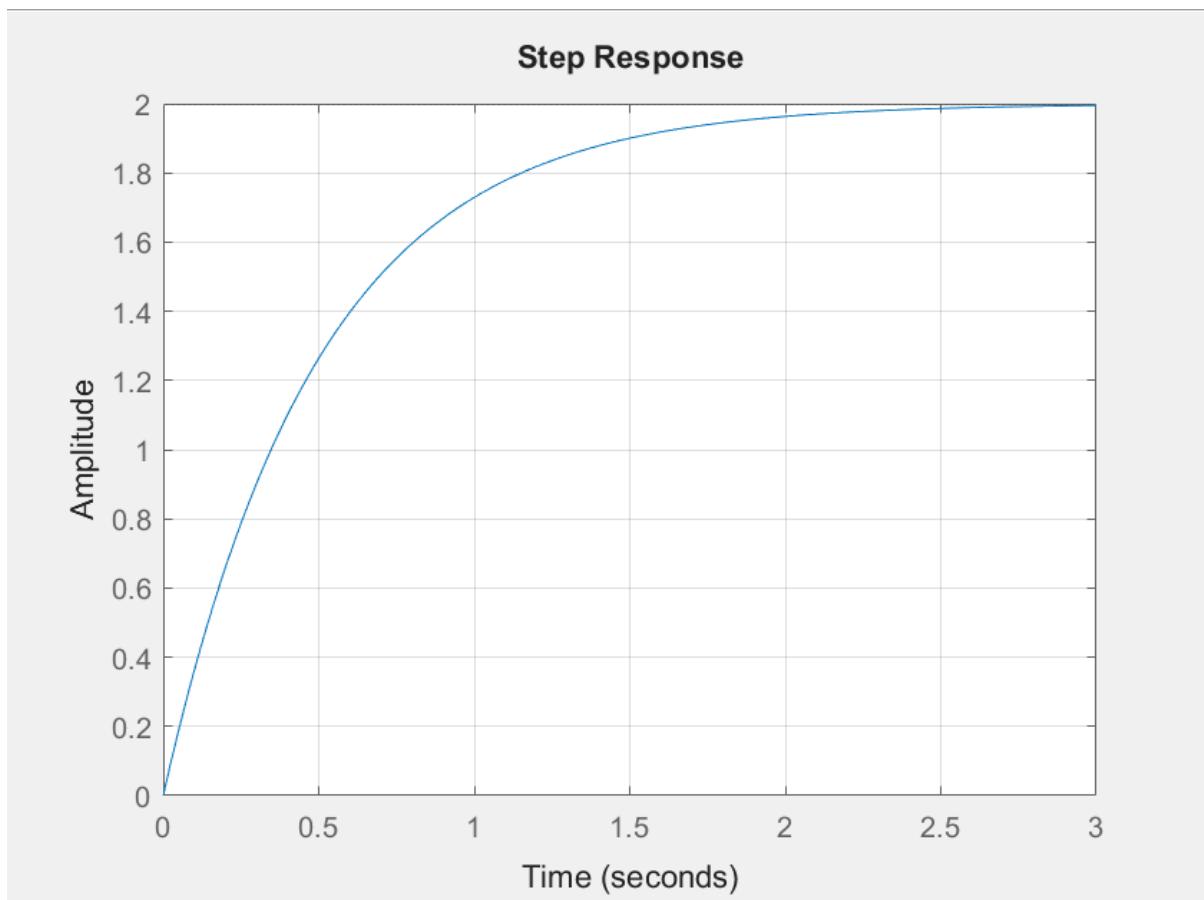
## Smart throttle Control

### Q1. Understanding a First-Order Plant Using Step Response

Consider the first-order plant:

$$G(s) = \frac{4}{s + 2}$$

1. Plot the unit step response in MATLAB.



2. Using the plot or `stepinfo`, determine:

- Time constant  $\tau$
- Rise time  $t_r$
- Settling time  $t_s$
- Final value (using Final Value Theorem)
- Steady-state error  $e_{ss}$

```
RiseTime: 1.0985
TransientTime: 1.9560
SettlingTime: 1.9560
SettlingMin: 1.8090
SettlingMax: 1.9987

Overshoot: 0
Undershoot: 0
Peak: 1.9987
PeakTime: 3.6611
```

Final Value=2

3. Compare MATLAB's final value with:

$$y_{ss} = \lim_{s \rightarrow 0} s \cdot G(s) \cdot \frac{1}{s}.$$

$$Y_{ss} = \lim_{s \rightarrow 0} \frac{4s}{s^2 + 2s} = \frac{4s}{2s} = 2$$

The steady state value is same as the DC gain of transfer function.

## Q2. System Type, Step Error, and Final Value Theorem

Given the plant:

$$G(s) = \frac{10}{s(s+5)}$$

- Identify the **system type** (count the number of integrators).

Type-1 system as there is only one integrator ( $s=0$ )

- Using the Final Value Theorem, find the steady-state error to a unit step:

$$e_{ss} = \lim_{s \rightarrow 0} s \left( \frac{1}{s} - G(s) \frac{1}{s} \right)$$

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2) b)  $e_{ss} = \lim_{s \rightarrow 0} s \left( \frac{1}{s} - \frac{G(s)}{s} \right)$

$$\lim_{s \rightarrow 0} (1 - G(s)) = (1 - \infty)$$

so it can be taken limit.

$\boxed{e_{ss} = 0}$  as multiplication

c) we consider closed-loop system  
(as it is unstable for an open-loop system)

$E(s) = R(s) - Y(s)$

$E(s) = \frac{Y(s)}{G(s)}$

$\frac{Y(s)}{G(s)} = R(s) - Y(s) \Rightarrow Y(s) \left( \frac{1+G(s)}{G(s)} \right) = R(s)$

$$T(s) = \frac{Y(s)}{R(s)} = \frac{G(s)}{1+G(s)}$$

Steady state value

- \* since  $E_{ss} = 0$ , the response must reach 1 exactly

### Overshoot

the open loop transfer function  $G(s)$

contains:

- \* an integrator: slows the rise, but increases overshoot

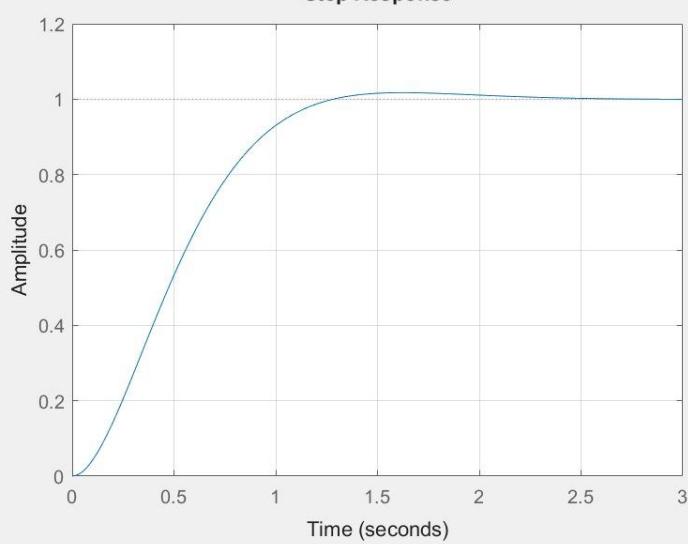
- \* stable pole at  $-s$ : gives moderate damping

for large gain  $K$ , root locus tends to cross imaginary axis  $\rightarrow$  overshoot likely

overshoot above 1 in step response

Settling  
as the dominant pole is close to zero, response will be slow to settle.

Step Response



### Q3. Required Specifications → Modify Transfer Function

Your goal is to design a first-order system that satisfies:

$$t_s < 1.2 \text{ seconds}, \quad e_{ss} = 0.1$$

1. Using the first-order formulas:

$$t_s \approx \frac{4}{a}, \quad e_{ss} = \frac{1}{1+K}$$

determine:

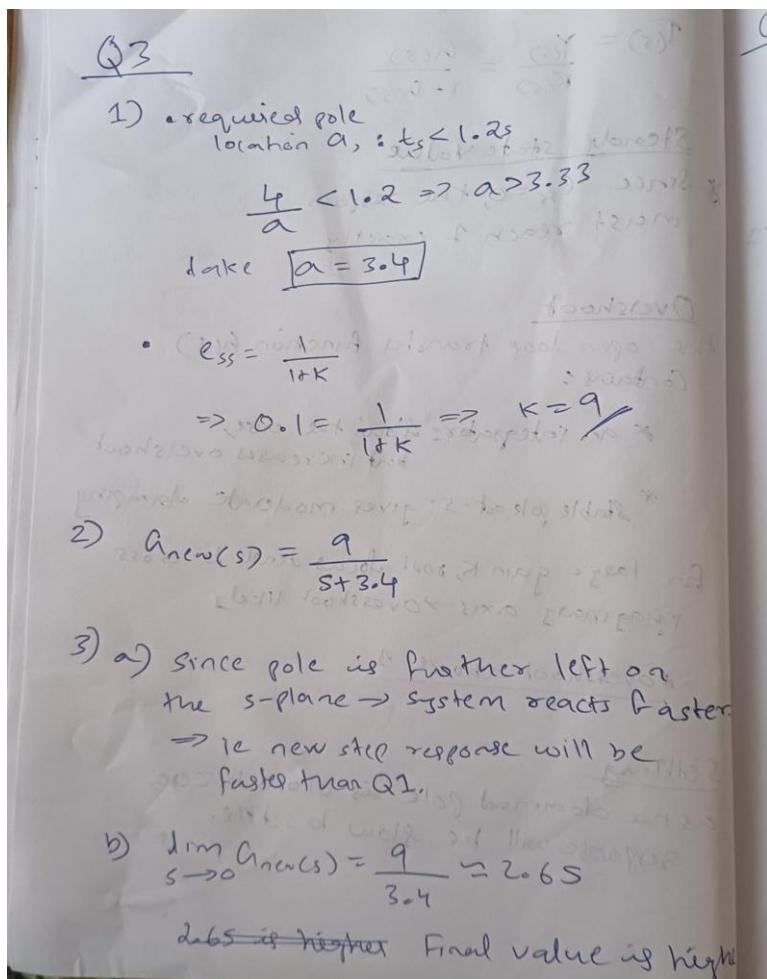
- the required pole location  $a$ ,
- the required static gain  $K$ .

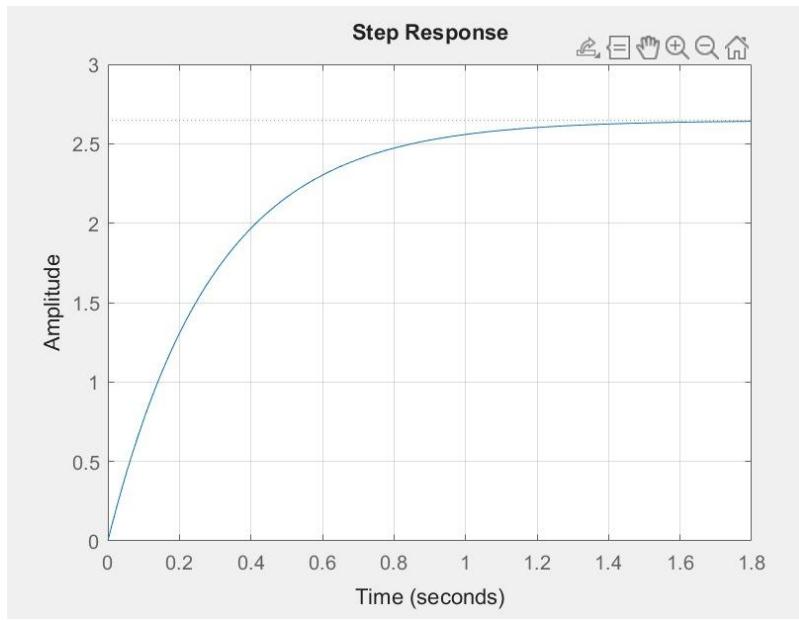
2. Construct the modified plant:

$$G_{\text{new}}(s) = \frac{K}{s+a}$$

3. Predict the shape of the step response before running MATLAB:

- Should it be faster or slower than Q1?
- Should the final value be higher or lower?





#### Q4. Designing a Simple Controller to Meet Specifications

You are given the following plant:

$$G(s) = \frac{3}{s+1}$$

You must design a simple controller:

$$C(s) = K(s + z)$$

to meet these desired characteristics:

$$t_s < 2 \text{ s}, \quad M_p < 10\%, \quad y_{ss} = 0.8$$

1. Using formulas from the class cheat sheet:

- Choose a zero  $z$  that reduces rise time.
- Choose a gain  $K$  that sets the desired steady-state value.
- Estimate the resulting damping ratio  $\zeta$  from the overshoot requirement.

Q4

1) \* a small positive zero ( $0 < z < 3$ ) reduces rise time.

\* a zero too close to origin adds overshoot.

So we choose  $z = 1$ .

$$* Y_{ss} = \lim_{s \rightarrow 0} T(s) \cdot \frac{(2)N(s)}{s} = (2)^{\frac{1}{2}} (s)$$

$$T(s) = \frac{C(s) G(s)}{1 + C(s) G(s)}$$

$$(C(s) = k z, G(s) = 3)$$

$$T(s) = \frac{3kz}{1 + 3kz} = 0.8 \text{ for } k=6, z=1$$

$$\Rightarrow 1 + 3kz = \frac{4}{3} \Rightarrow k = \frac{4}{3}$$

Overshoot

$$M_p = e^{\frac{-C\pi}{5(1-C^2)}}$$

$$M_p < 0.10$$

$$0.1 = e^{-\frac{C\pi}{\sqrt{1-C^2}}}$$

$$\Rightarrow \ln(0.1) = \frac{-C\pi}{\sqrt{1-C^2}}$$

$$\Rightarrow \frac{C\pi}{\sqrt{1-C^2}} = 2.302$$

$$\Rightarrow C \approx 0.59$$

moderate damping  
small overshoot.

$$2) T(s) = \frac{(Cs)H(s)}{1+(Cs)H(s)} = \frac{3K}{1+3K}$$

3)

- a) adding the zero slightly increases overshoot but < 10%, as we chose small  $Z=1$ , so overshoot should remain controlled.

b)

$$g_{ss} = \frac{3Kz}{1+3Kz}$$

as  $K$  increase numerator increases.  
 $\rightarrow$  increase in final value.

- c) zero gain will push effective pole leftward.

$\rightarrow$  Response will be faster.

## Q5. Ramp Tracking and System Type

Using the controller and closed-loop system from Q4:

$$r(t) = t \quad (\text{unit ramp})$$

1. Determine the **system type** of the closed-loop system.
2. Using system type rules, predict whether the ramp error will be:
  - infinite,
  - finite non-zero,
  - or zero.
3. Verify using the Final Value Theorem for ramp input:

$$e_{ss}^{\text{ramp}} = \lim_{s \rightarrow 0} s \left( \frac{1}{s^2} - T(s) \frac{1}{s^2} \right).$$

4. Explain whether adding the zero at  $(s + z)$  helps or hurts ramp tracking.

Q5 Shows  $s \rightarrow \infty$  be a good approach (P)

1)  $r(t) = t \xrightarrow{\mathcal{L}} R(s) = \frac{1}{s^2}$

open loop transfer function  $= L(s) = C(s) G(s)$   
 has no factor of  $\frac{1}{s}$   $\Rightarrow$  system type = 0

2) From system type rule, ramp error will be infinite.

3)  $e_{ss} = \lim_{s \rightarrow 0} s \left[ \frac{1}{s^2} - \frac{T(s)}{s^2} \right]$   
 $= \lim_{s \rightarrow 0} s \left[ \frac{1}{s^2} - \frac{0.8}{s^2} \right] = \lim_{s \rightarrow 0} \frac{0.2}{s} \xrightarrow{\frac{0.2}{s} \rightarrow \infty}$

$\Rightarrow$  Final value theorem validates infinite ramp error.

4) adding a zero at  $s = -2$  cancels plant pole at  $s = -1$ .

↳ this removes system's dynamic response, only resulting in constant gain.

\* for good ramp tracking you need:

\* one integrator in open loop  
↳ gives finite ramp errors.

So adding zero hurts ramp tracking

because it makes system type 0, giving infinite ramp errors.