



**University of
New Haven**

PROJECT 2

MECH – 6627

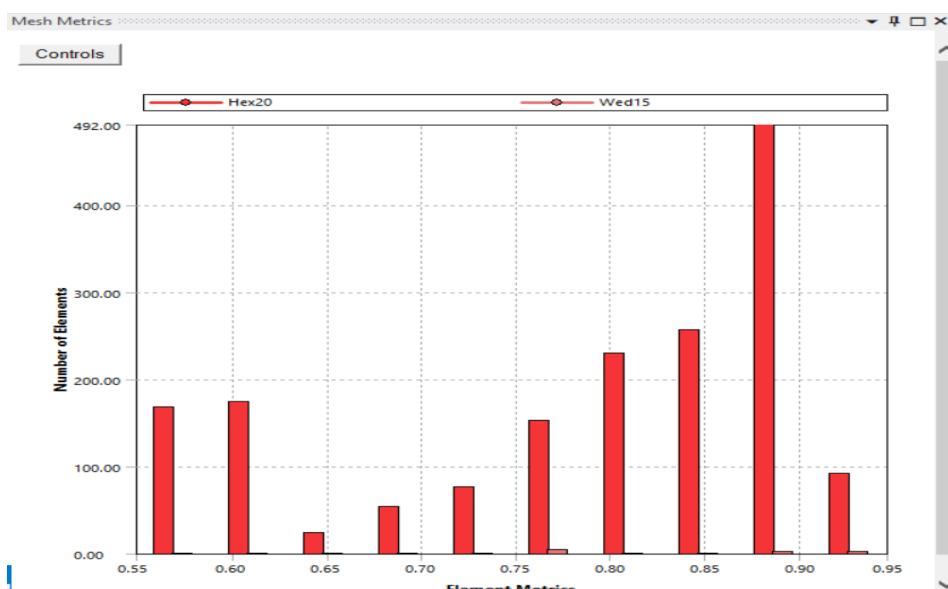
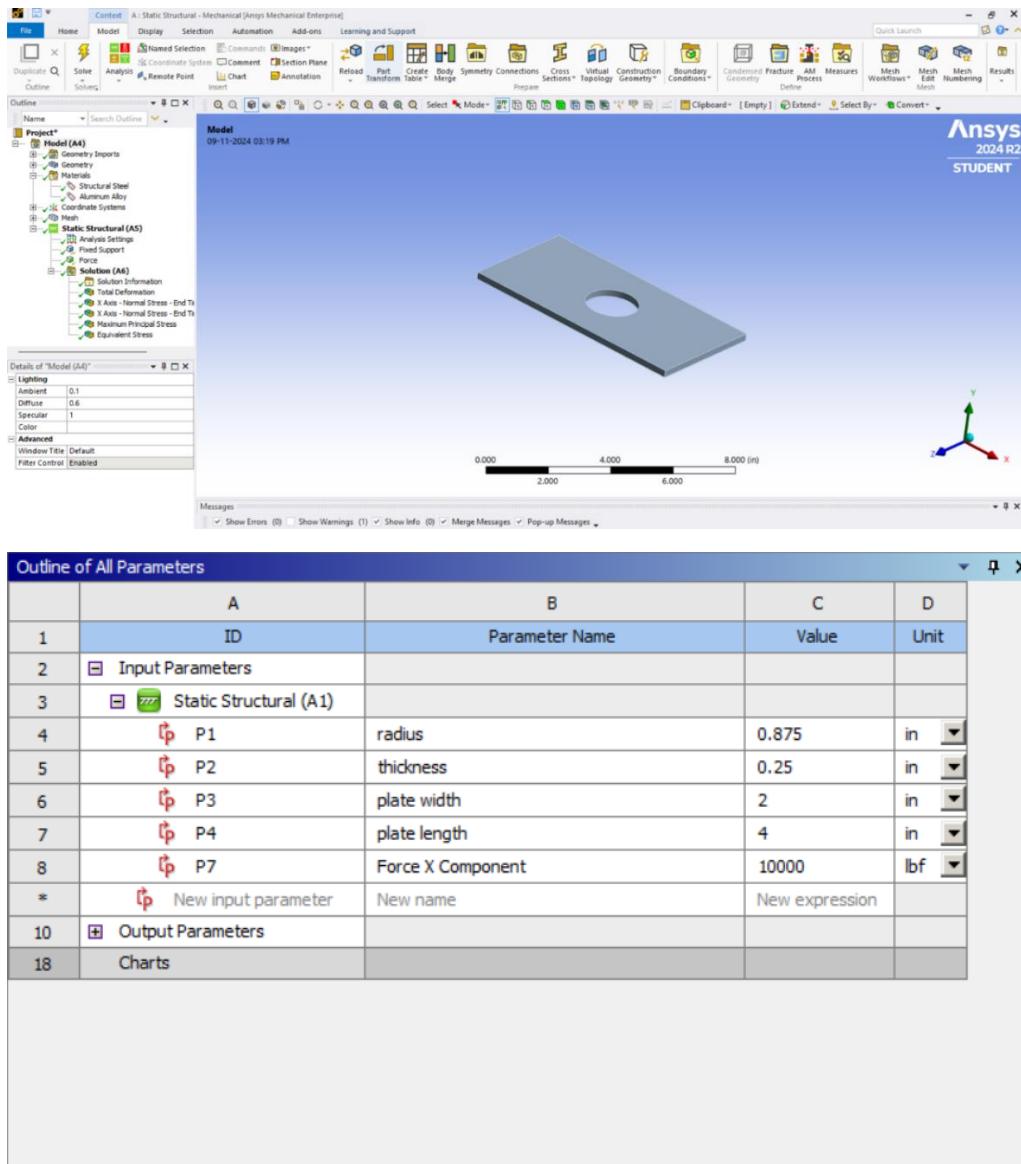
**Computer Aided
Engineering**

Professor – George Bauer

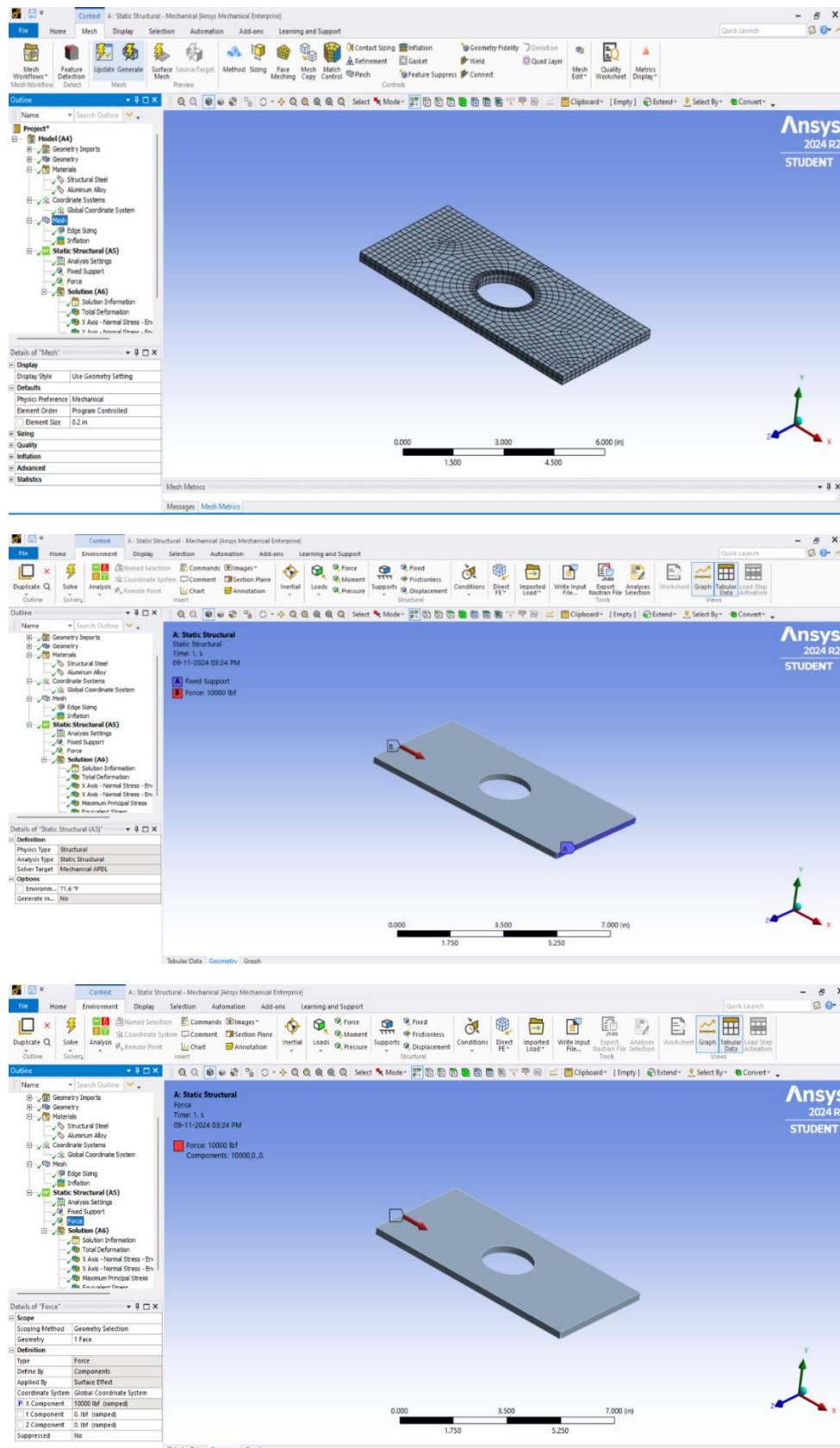
By

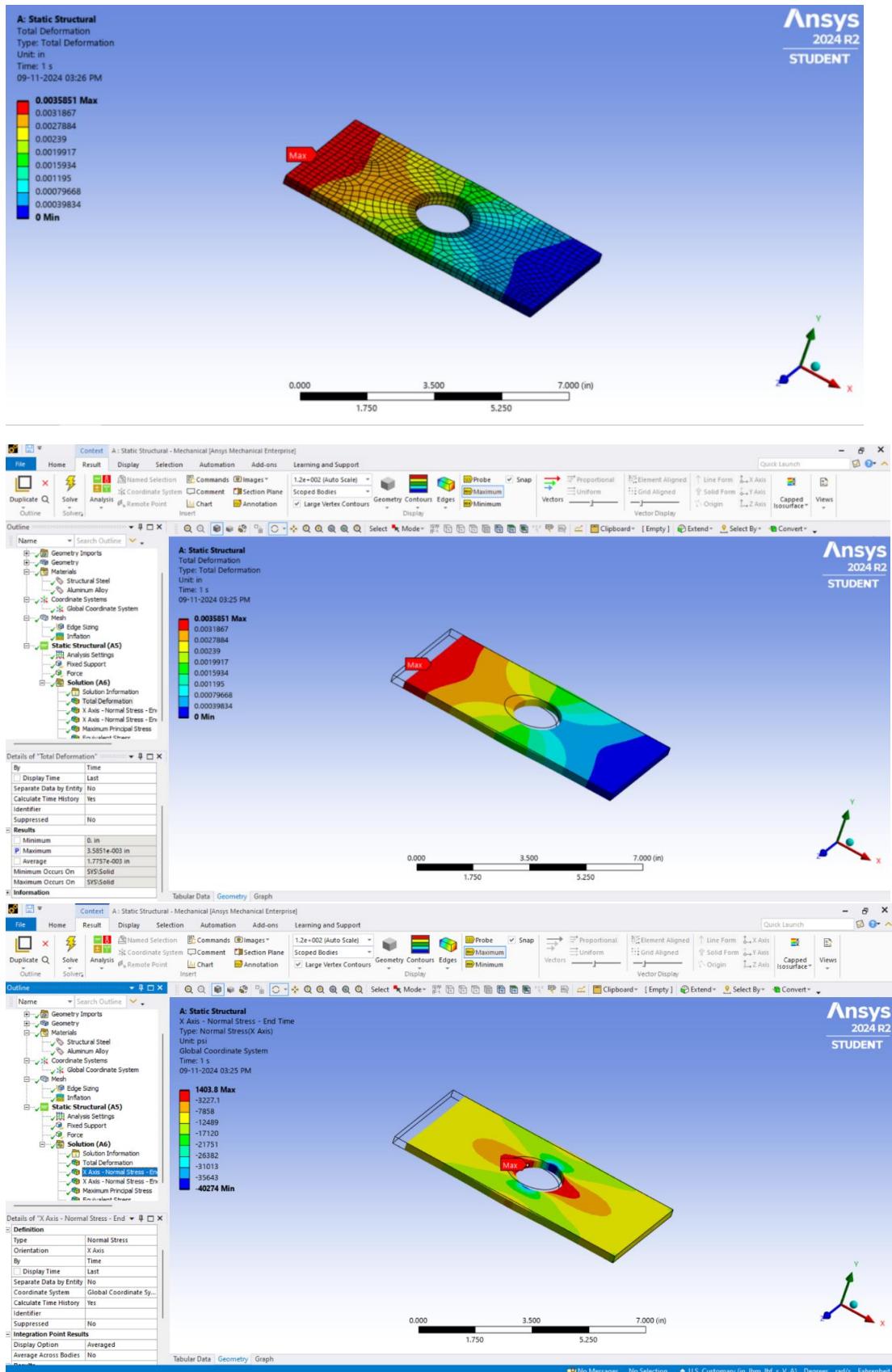
Yashwanth Kanthala (00921675)

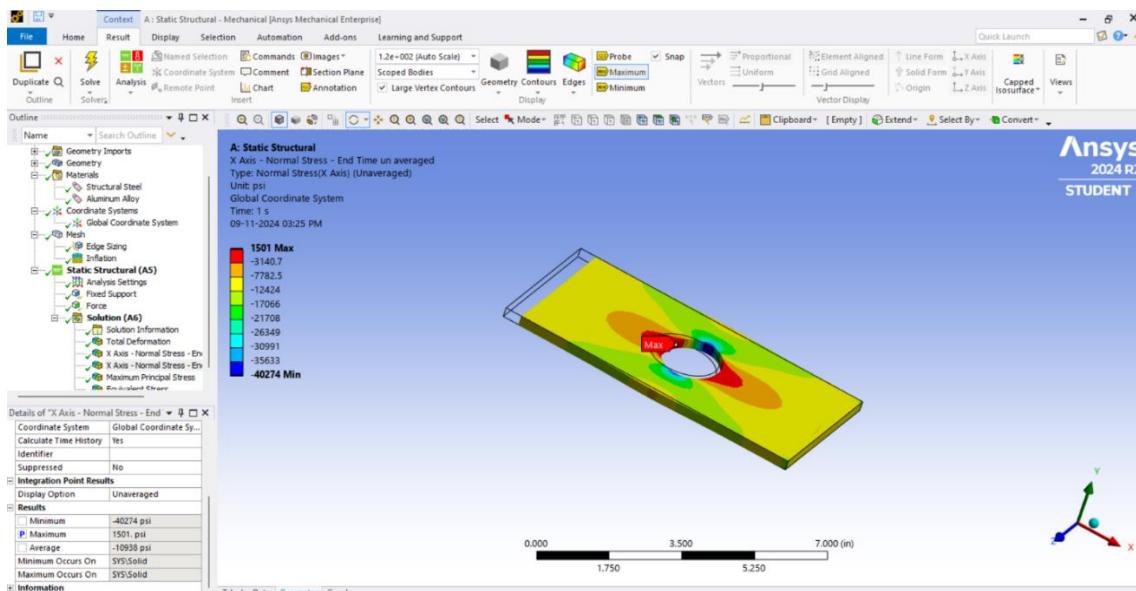
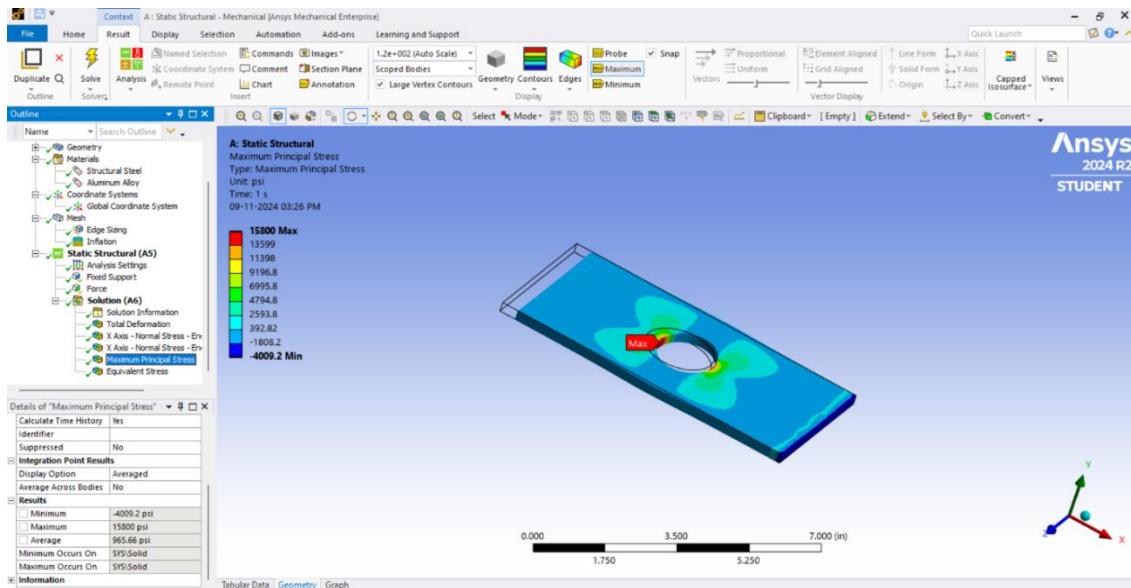
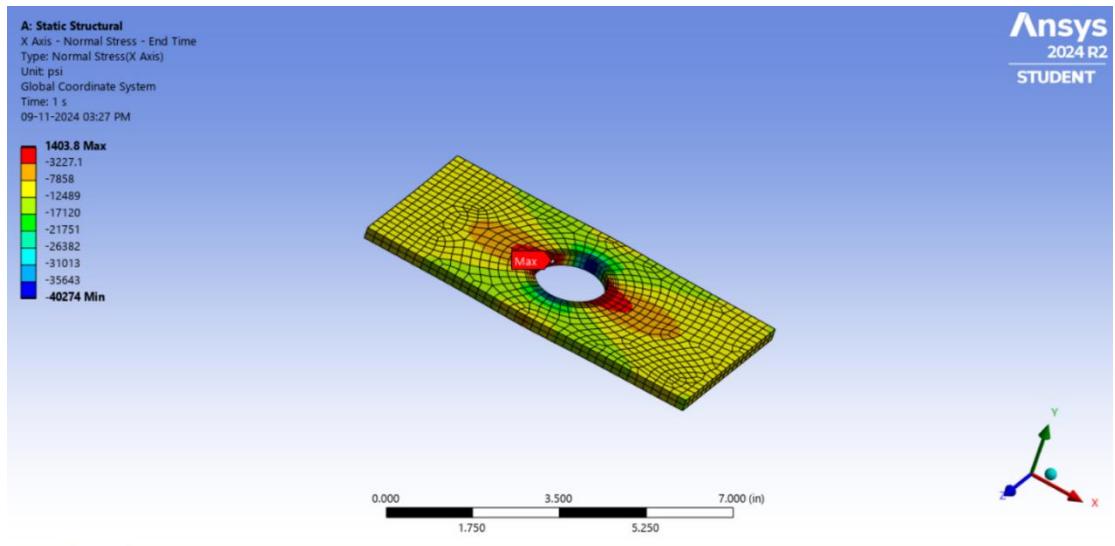
TASK1: EXAMPLE 1

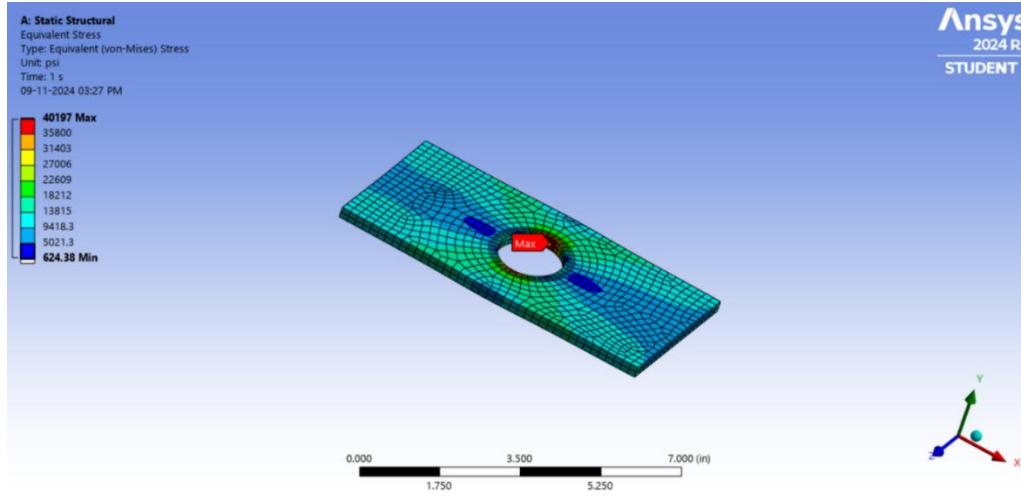
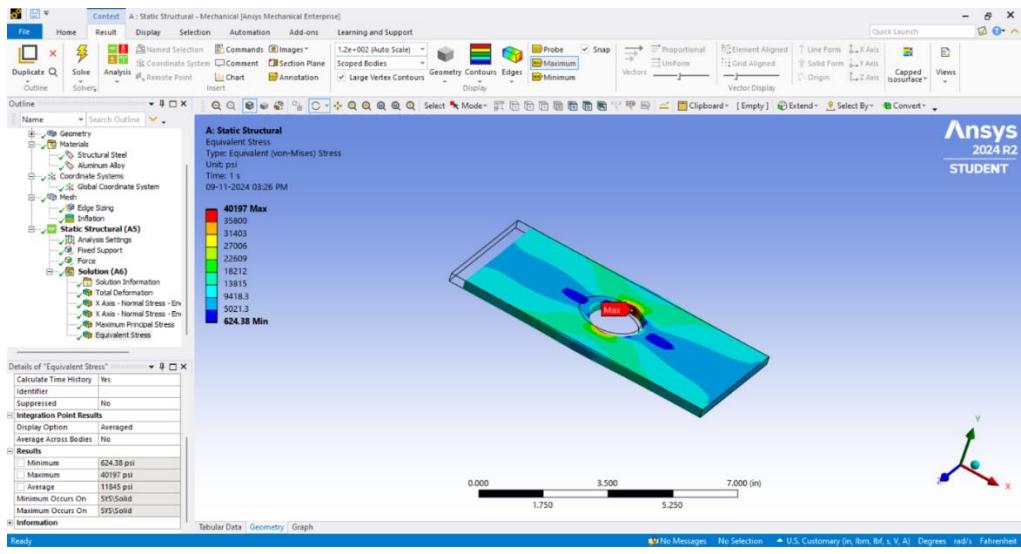


MESHING









CALCULATION FOR TASK 1

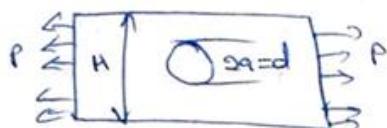
length = 8 inch

Applied load = 10000 lbs

width = 4 inch

Diameter = 1.75 inch.

Area of rectangle plate



$$A = (b - d)h$$

$$A \Rightarrow (4 - 1.75) \times 0.25$$

$$= 2.25 \times 0.25$$

$$= 0.5625 \text{ inch}^2$$

$$\sigma_{\text{nom}} \Rightarrow \frac{P}{A} \Rightarrow \frac{10000 \text{ lbs}}{0.5625} = 16200 \text{ psi}$$

stress contraction factor

$$\frac{D}{d} \Rightarrow \frac{4}{1.75} = 2.28$$

from table the radius ratio is 2.88 and

K_t value is 2.5

$$\sigma_{\text{max}} = K_t \times \sigma_{\text{nominal}}$$

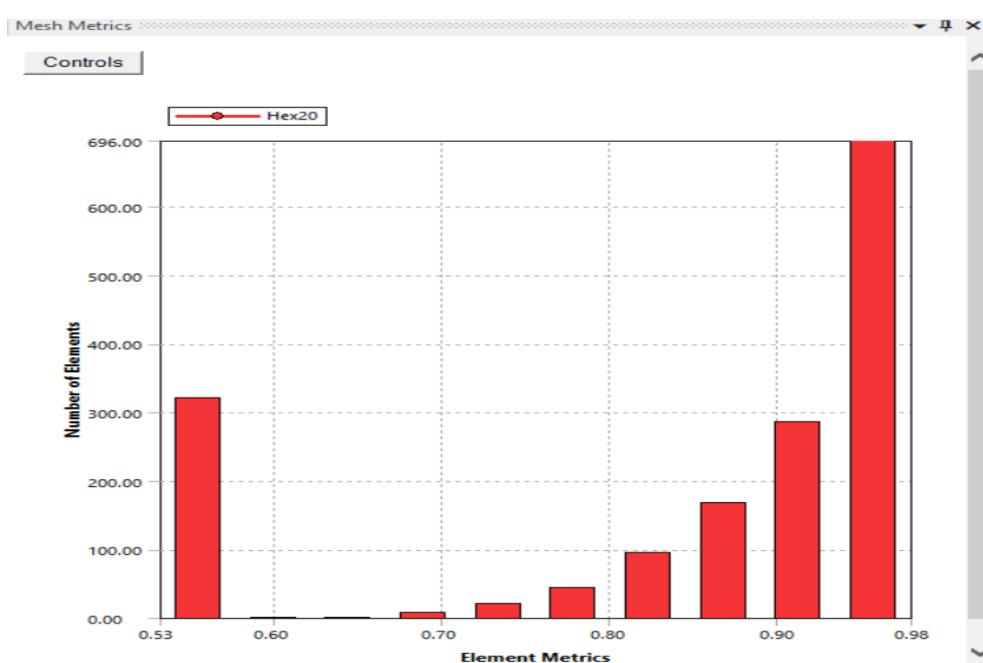
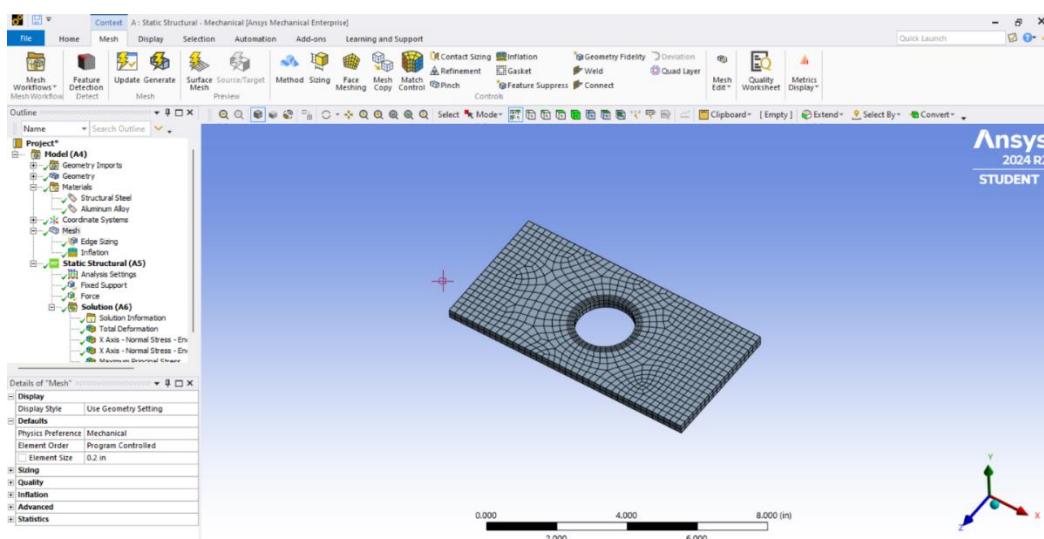
$$\sigma_{\text{max}} = 2.5 \times 16200$$

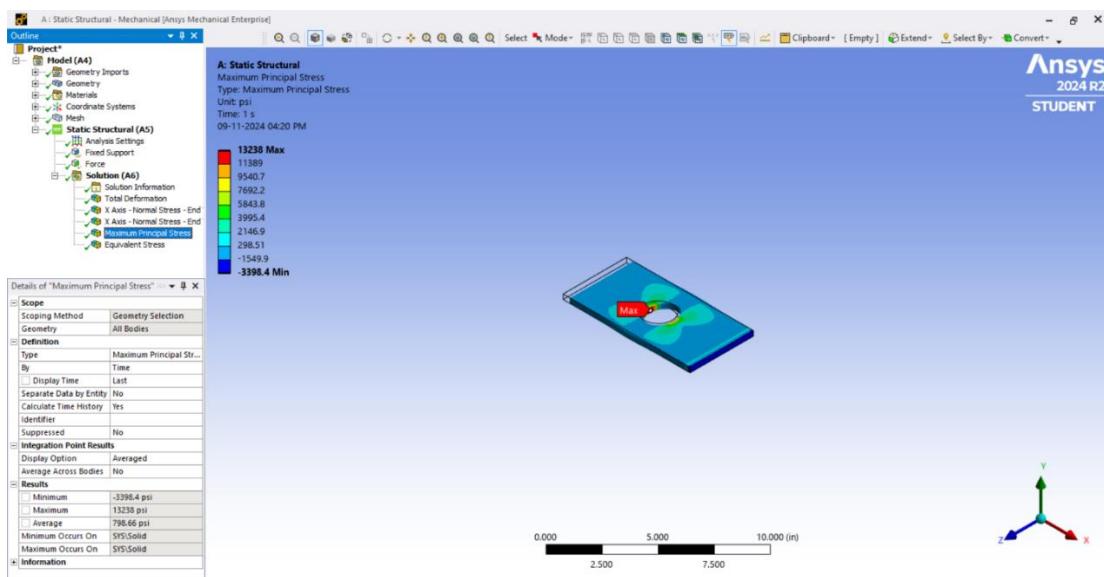
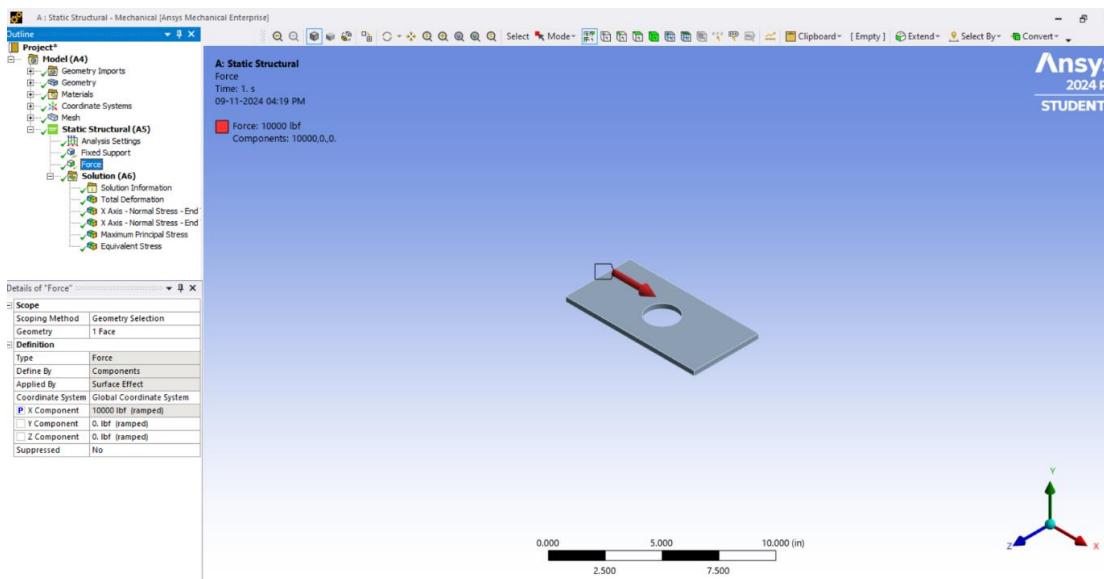
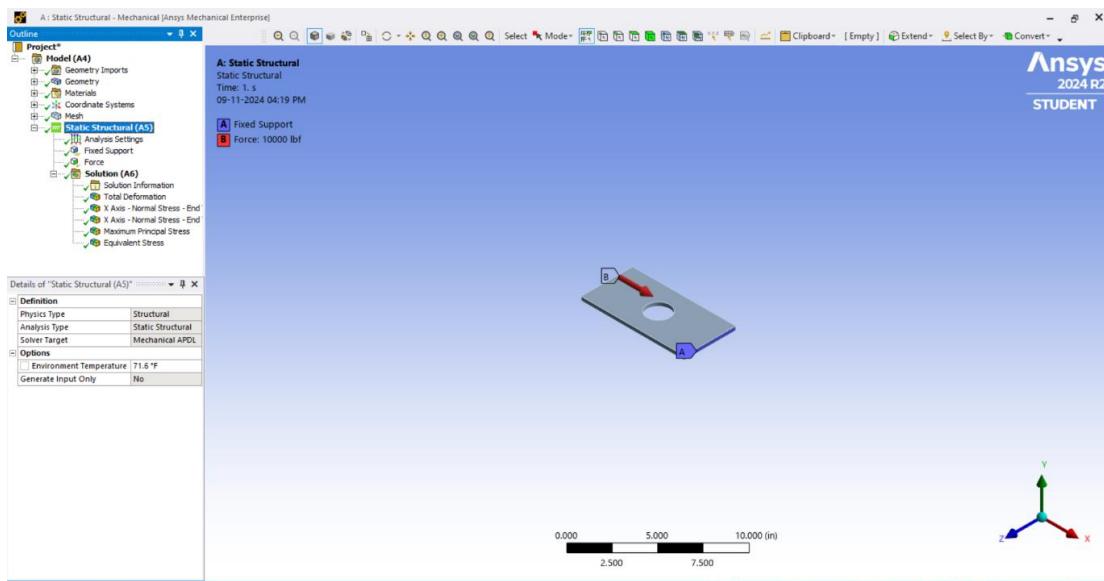
$$= 40500 \text{ psi}$$

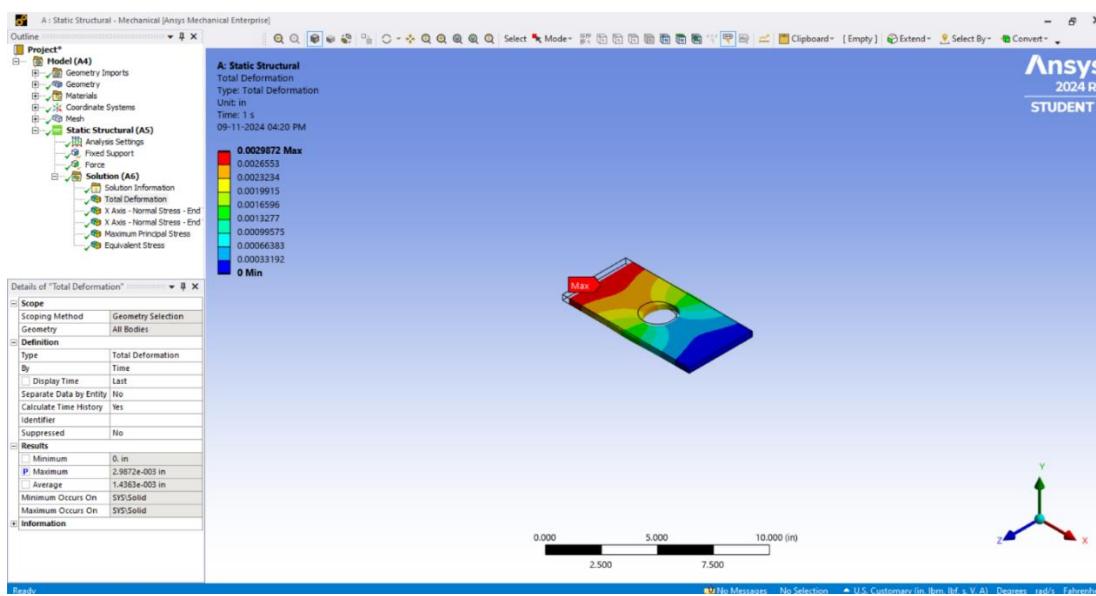
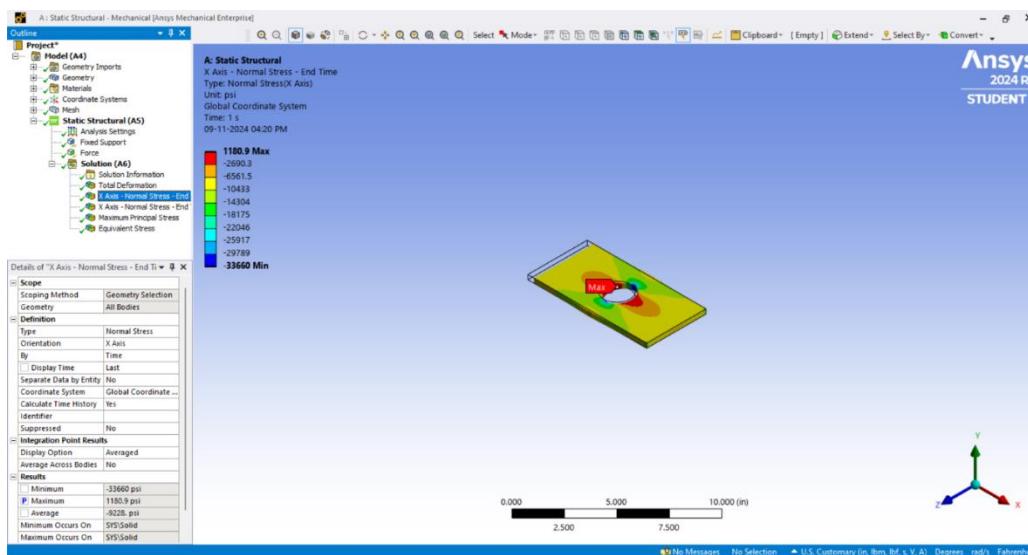
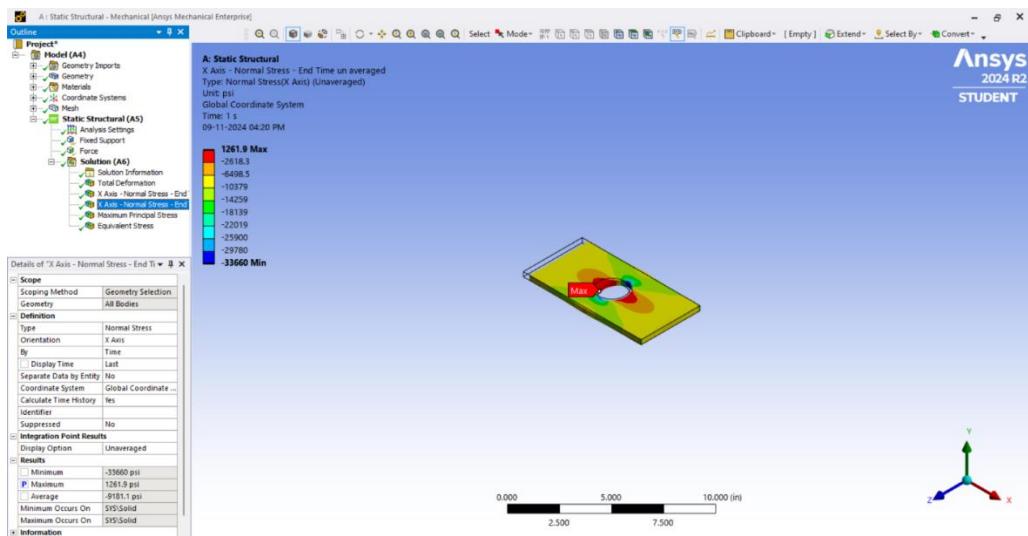
In reality, rectangular plate the material properties, hole diameter and thickness of the plate plays an important role in finding the stress. In the given fig shows a good mesh in the edges of the holes. The stress acting at the hole will show good deformation. If the area of the hole is larger than the rectangular plate in the practical life it is not possible as the stress acts on the edges of its shows large deformations and plate will not withstands for the hole diameter.

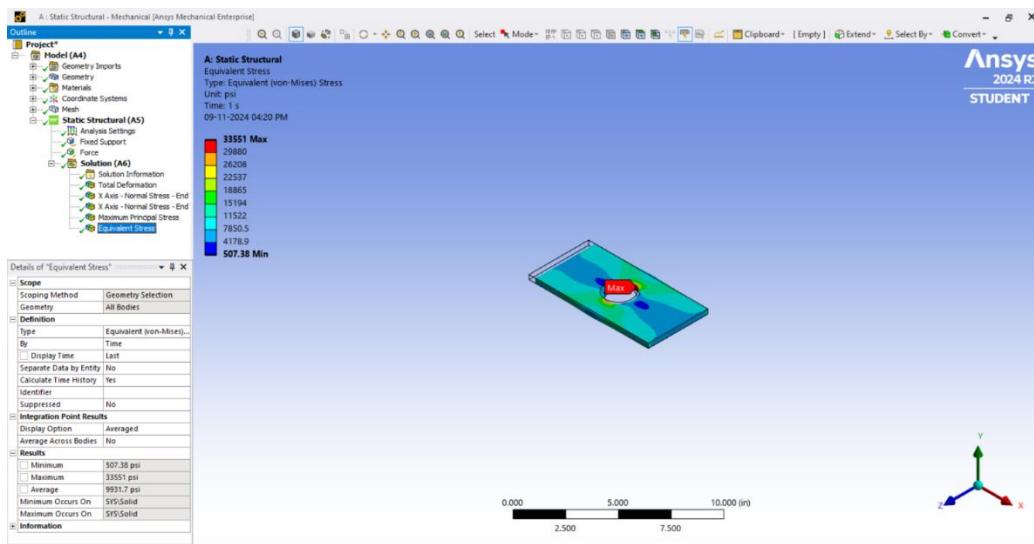
Example 2

	A	B	C	D
1	ID	Parameter Name	Value	Unit
2	Input Parameters			
3	Static Structural (A1)			
4	P1	radius	0.875	in
5	P2	thickness	0.25	in
6	P3	plate width	3	in
7	P4	plate length	4	in
8	P7	Force X Component	10000	lbf
*	New input parameter			
10	Output Parameters			
18	Charts			





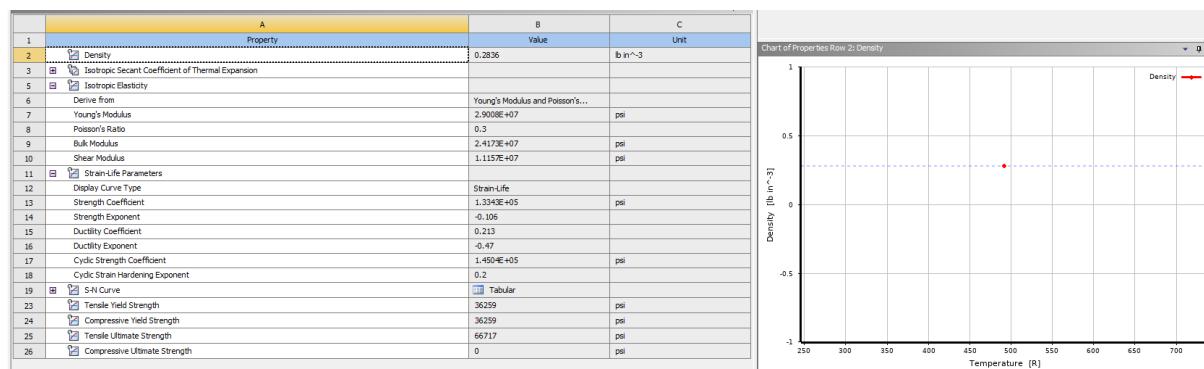




ALUMINUM ALLOY

Properties of Outline Row 4: Aluminum Alloy			
	A	B	C
1	Property	Value	Unit
2	Density	0.10007	lb in^-3
3	Isotropic Secant Coefficient of Thermal Expansion		
5	Isotropic Elasticity		
6	Derive from	Young's Modulus and Poisson's...	
7	Young's Modulus	1.0298E+07	psi
8	Poisson's Ratio	0.33	
9	Bulk Modulus	1.0096E+07	psi
10	Shear Modulus	3.8713E+06	psi
11	S-N Curve	Tabular	
15	Tensile Yield Strength	40611	psi
16	Compressive Yield Strength	40611	psi
17	Tensile Ultimate Strength	44962	psi
18	Compressive Ultimate Strength	0	psi

STRUCTRAL STEEL



EXAMPLE 2

2) Radius = 0.85 inch.

$$D = 1.75 \text{ inch}$$

thickness = 0.2 inch.

width = 6

length = 8.

$$\text{Area} = (b-d)t \Rightarrow (6 - 1.75) \times 0.3$$

$$\Rightarrow 4.25 \times 0.3 = 0.85 \text{ inch}^2$$

$$\sigma_{\text{normal}} \Rightarrow \frac{P}{A} \Rightarrow \frac{10000}{0.85} = 11764.706 \text{ psi}$$

stress contraction factor K_t

$$\frac{W}{d} \Rightarrow \frac{6}{1.75} = 3.4$$

from table the fillet radius ratio is

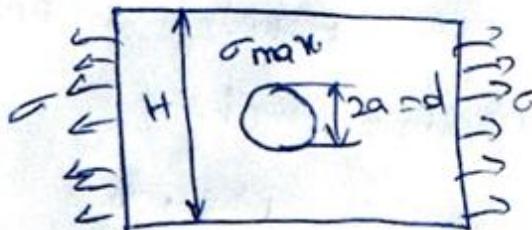
3.4 and K_t value is 2.8

maximum stress σ_{max} at hole

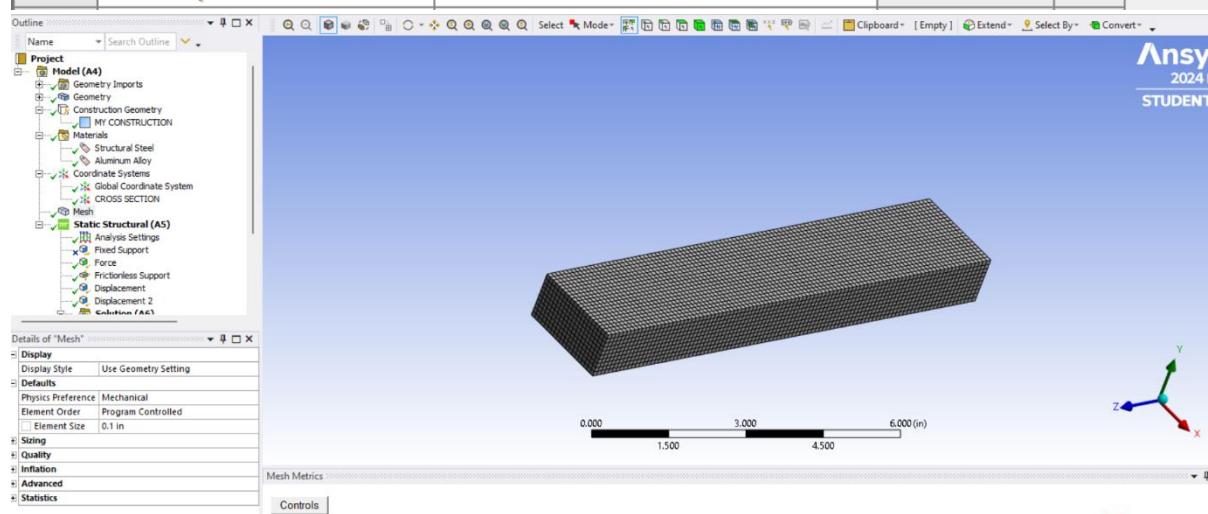
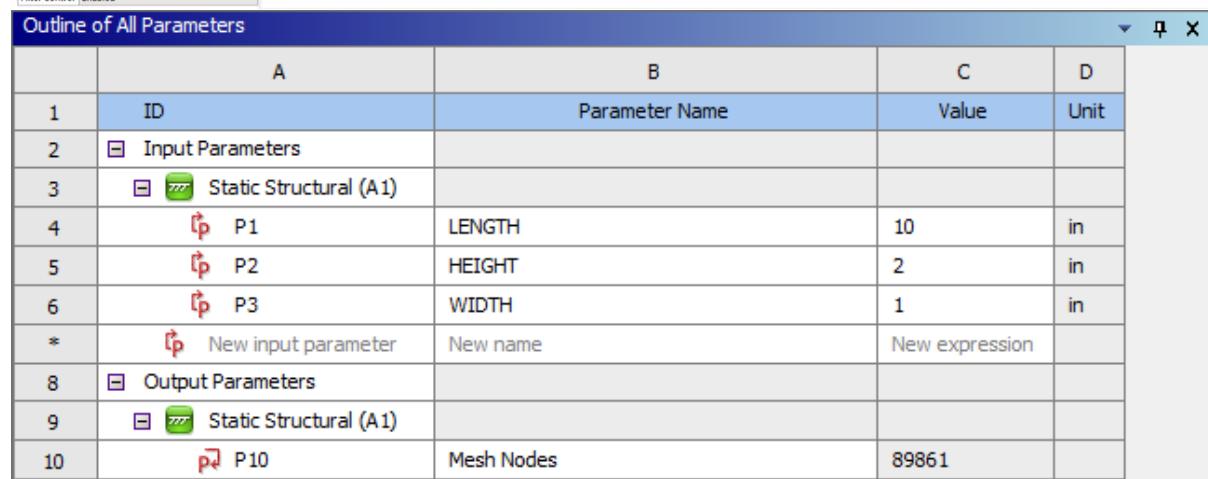
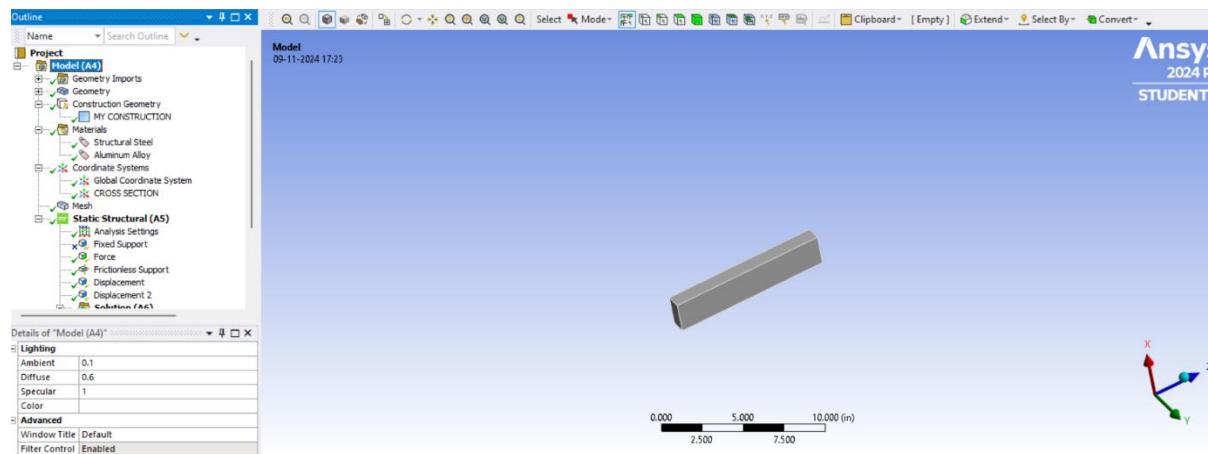
$$\sigma_{\text{max}} \Rightarrow K_t \times \sigma_{\text{normal}}$$

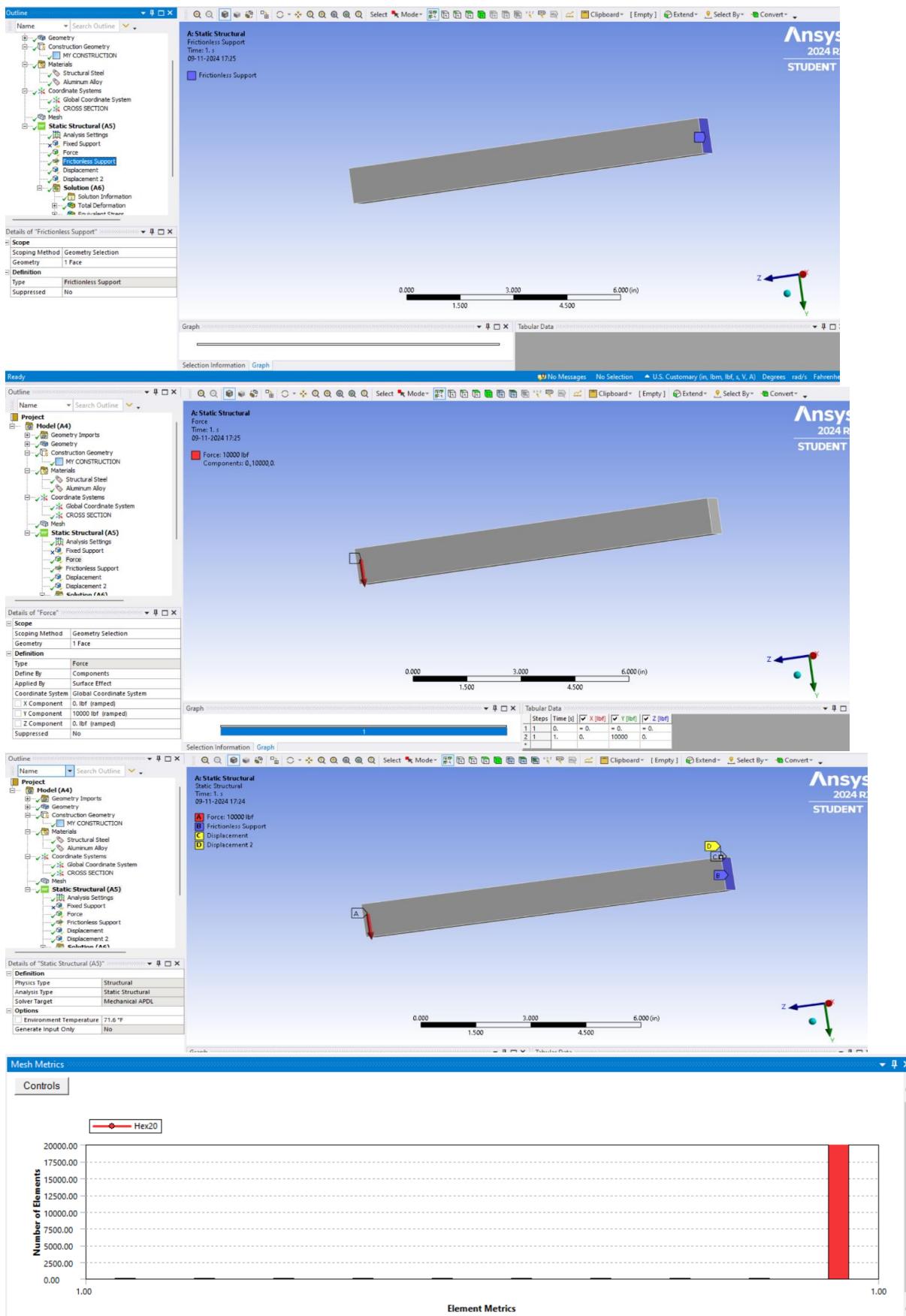
$$\Rightarrow 2.8 \times 11764.706$$

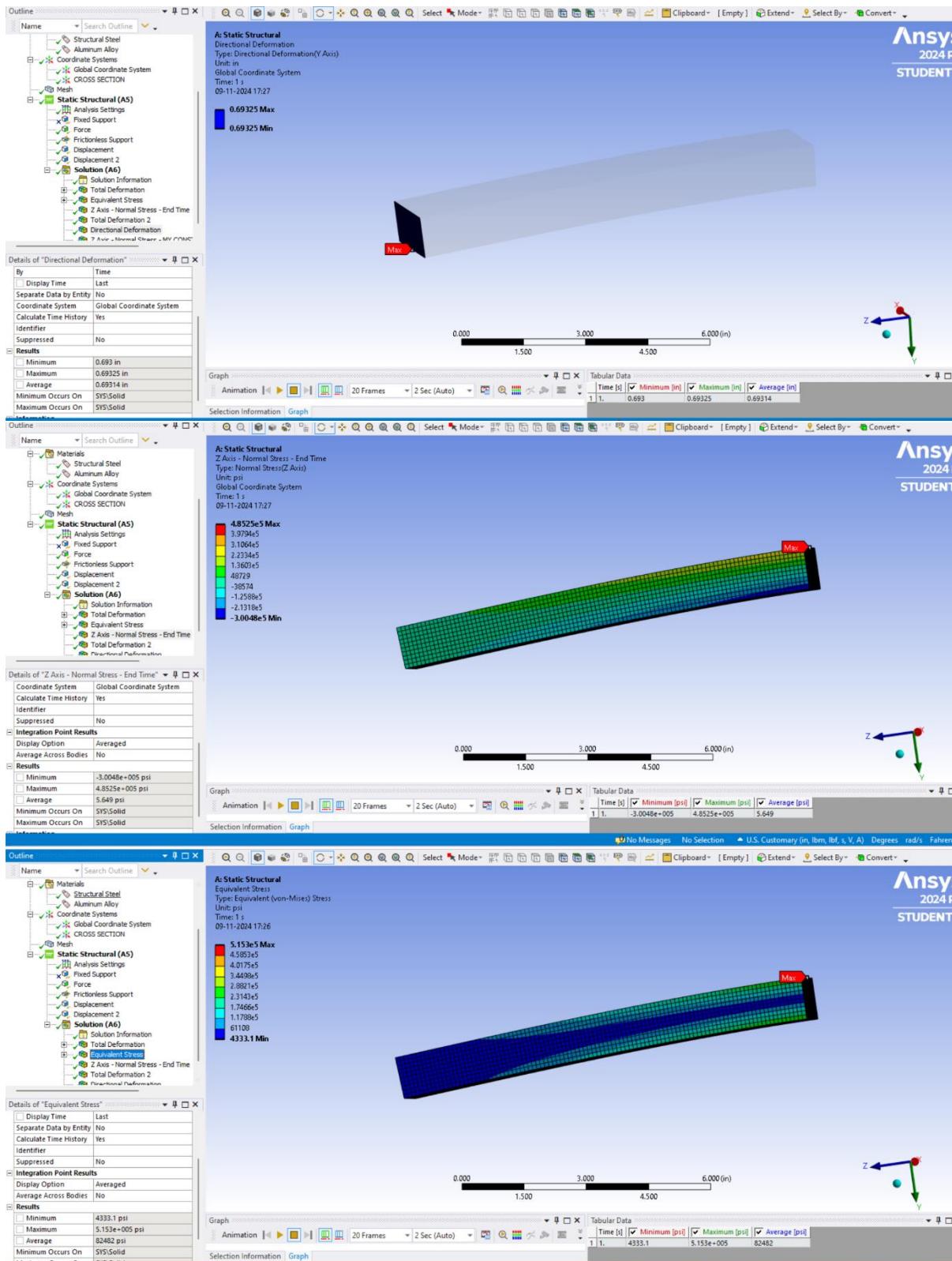
$$\Rightarrow 32939.2 \text{ psi}$$

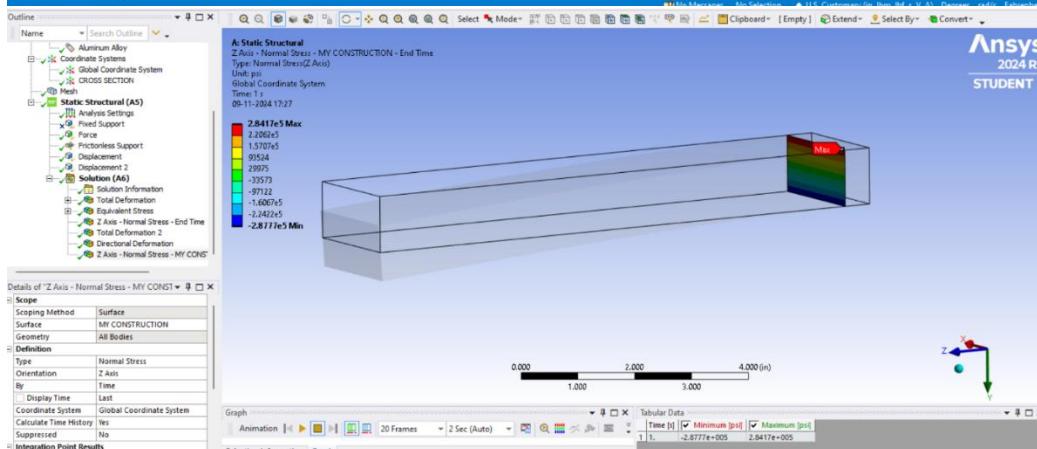
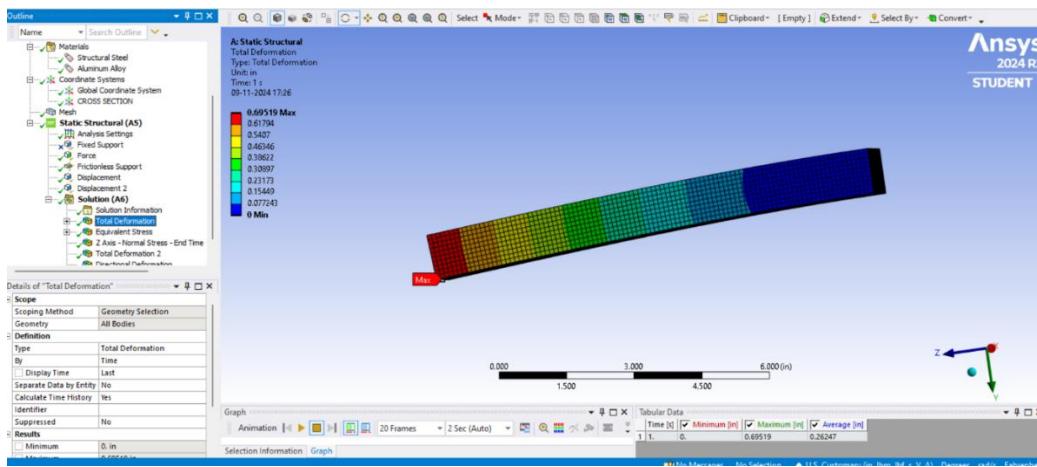


Task 2









	A	B	C
	Property	Value	Unit
1			
2	Density	0.10007	lb in^-3
3	Isotropic Secant Coefficient of Thermal Expansion		
5	Isotropic Elasticity		
6	Derive from	Young's Modulus and Poisson's...	
7	Young's Modulus	1.0298E+07	psi
8	Poisson's Ratio	0.33	
9	Bulk Modulus	1.0096E+07	psi
10	Shear Modulus	3.8713E+06	psi
11	S-N Curve	Tabular	
15	Tensile Yield Strength	40611	psi
16	Compressive Yield Strength	40611	psi
17	Tensile Ultimate Strength	44962	psi
18	Compressive Ultimate Strength	0	psi

EXAMPLE 1

$$2) \text{ thickness} = 1 \text{ inch}$$

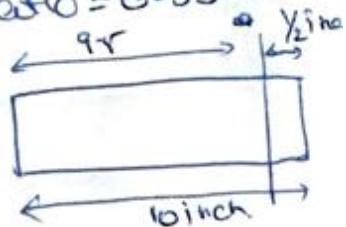
$$\text{height} = 2 \text{ inch.}$$

$$\text{length} = 10 \text{ inch}$$

$$I = \frac{1}{12} h^3 b \Rightarrow \frac{1}{12} \times (2)^3 \times 1 \\ = \frac{1}{12} \times 8 \times 1 = 0.667$$

$$\text{young's modulus} = 1.029 \times 10^7$$

$$\text{poisson's ratio} = 0.33$$



$$M \Rightarrow \cancel{10000} \cdot PXL$$

$$\Rightarrow 10000 \times (10 - \gamma_2) \\ = 10000 \times 9.5 = 95000$$

$$C = \frac{1}{2} \times h = \frac{1}{2} \times 2 = 1$$

$$\sigma_c \Rightarrow \frac{MC}{I} \Rightarrow \frac{95000 \times 1}{0.667} = 14.423 \times 10^5 \text{ psi}$$

$$\text{from table } k_t = 2.4$$

$$\sigma_{\max} \Rightarrow k_t \times 1.423 \times 10^5 \\ \Rightarrow 2.4 \times 1.423 \times 10^5 \\ 34152 \text{ psi}$$

$$\gamma = \frac{PL^3}{3EI} = \frac{10000 \times (10)^3}{3 \times 1.029 \times 10^7 \times 0.667}$$

$$\gamma \Rightarrow 0.04856727$$

The bending stress of the rectangular bar is determined by manually as well as ansys shows a minimal differences so the rectangular bar can withstand in the stress . the mesh shows goods so we can find the deformation half inch away from the edges shows at the surfaces shows the rectangular bar can withstand for given loads and shows a small deformation in the edges.

EXAMPLE 2

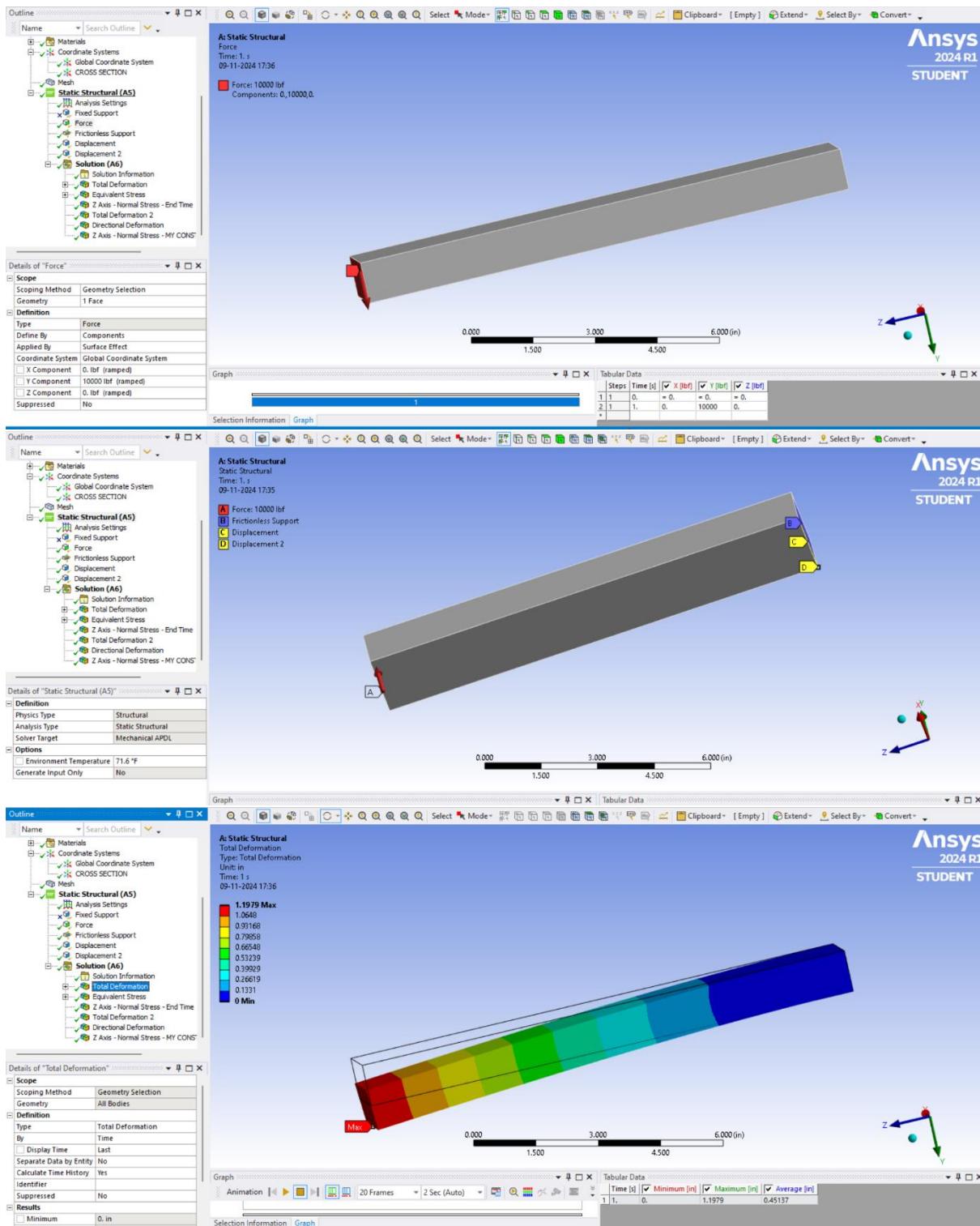
The following screenshots illustrate the Ansys Mechanical Enterprise interface during the analysis setup for Example 2.

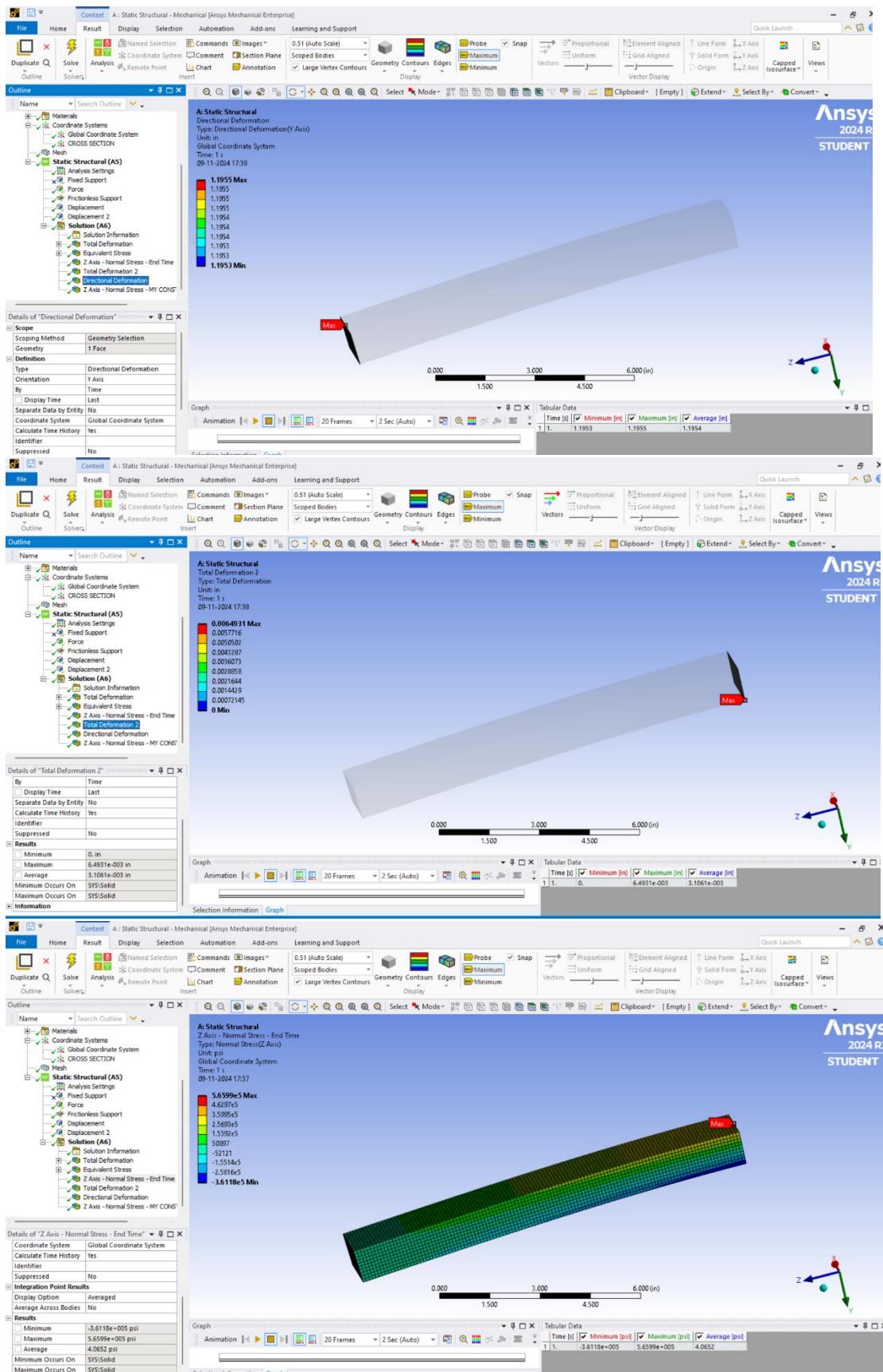
Top Screenshot: Shows the main Ansys Mechanical Enterprise window with the "Static Structural (A5)" analysis type selected. The geometry is a rectangular bar meshed with hexagonal elements. A coordinate system is shown at the bottom right.

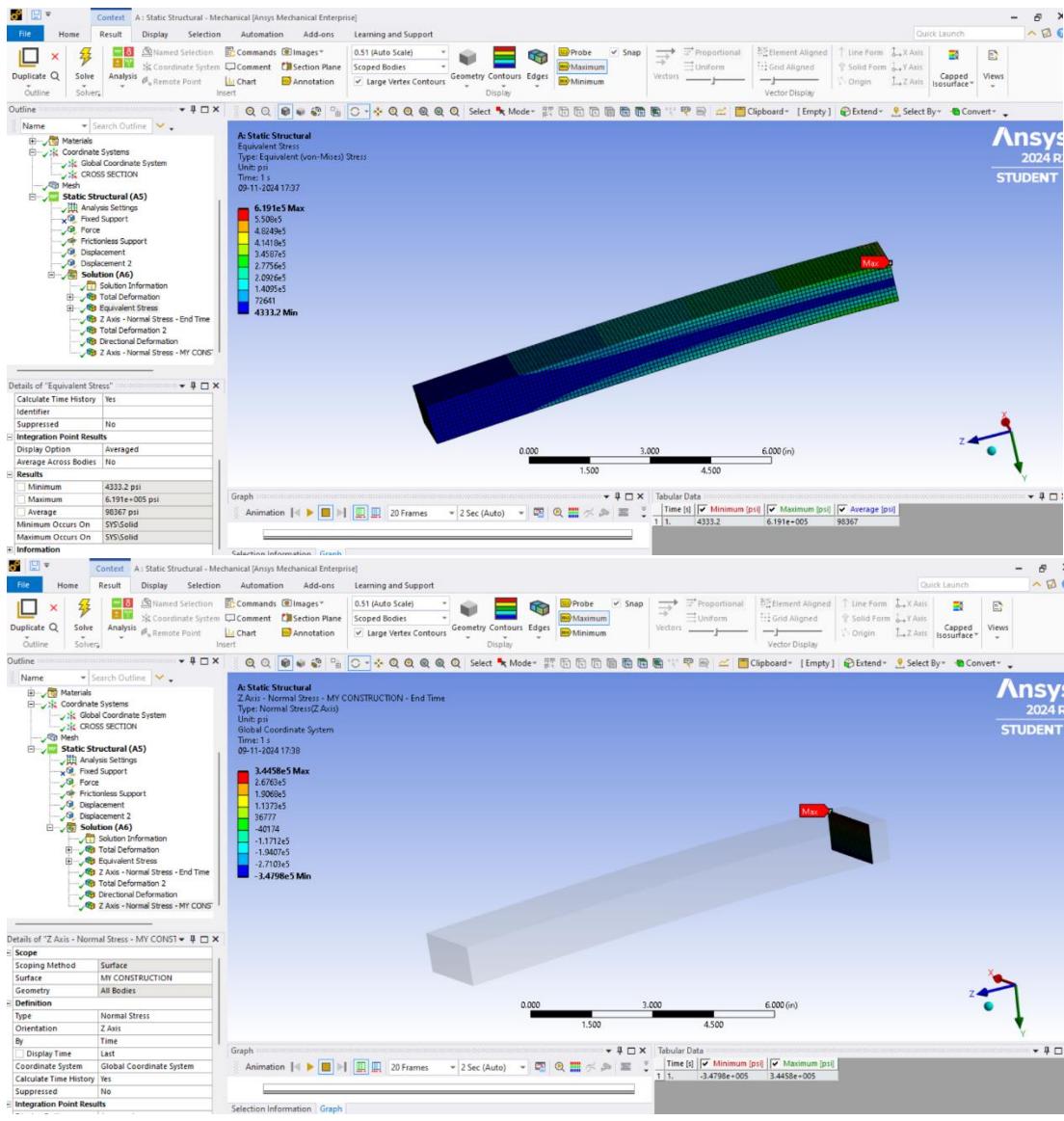
Outline of All Parameters:

	A	B	C	D
1	ID	Parameter Name	Value	Unit
2	Input Parameters			
3	Static Structural (A1)			
4	P1	LENGTH	12	in
5	P2	HEIGHT	2	in
6	P3	WIDTH	1	in
*	New input parameter			
8	Output Parameters			
9	Static Structural (A1)			
10	P10	Mesh Nodes	1.1615E+05	

Bottom Screenshot: Shows the completed setup with a larger rectangular bar meshed with hexagonal elements. The coordinate system is shown at the bottom right. The "Mesh Metrics" plot shows a distribution of element sizes, with most elements having a metric of 1.00 and a few outliers near 1.10.







Properties of Outline Row 4: Aluminum Alloy			
	A	B	C
	Property	Value	Unit
1	Density	0.10007	lb in^-3
2	Isotropic Secant Coefficient of Thermal Expansion		
3	Isotropic Elasticity		
6	Derive from	Young's Modulus and Poisson's...	
7	Young's Modulus	1.0298E+07	psi
8	Poisson's Ratio	0.33	
9	Bulk Modulus	1.0096E+07	psi
10	Shear Modulus	3.8713E+06	psi
11	S-N Curve	Tabular	
15	Tensile Yield Strength	40611	psi
16	Compressive Yield Strength	40611	psi
17	Tensile Ultimate Strength	44962	psi
18	Compressive Ultimate Strength	0	psi

EXAMPLE 2

thickness = 1 inch

Length = 12 inch

height \Rightarrow 2 inch

Young's module = 1.029×10^7

Poisson Ratio = 0.33

$$I \Rightarrow \frac{1}{12} h^3 x b = \frac{1}{12} \times (2)^3 = \frac{1}{12} \times 8 \times 1 = 0.667$$

$$M_c = P \times L \Rightarrow 10000 \times (12 - 0.5) \Rightarrow 10000 \times 11.5$$

$$\Rightarrow 115000 \text{ psi}$$

$$c = \frac{1}{2} h \Rightarrow \frac{1}{2} \times 2 = 1$$

$$\sigma_{\text{nominal}} \Rightarrow \frac{M_c}{I} \Rightarrow \frac{115000 \times 1}{0.667} = 1.724 \times 10^5$$

Kt value from table =

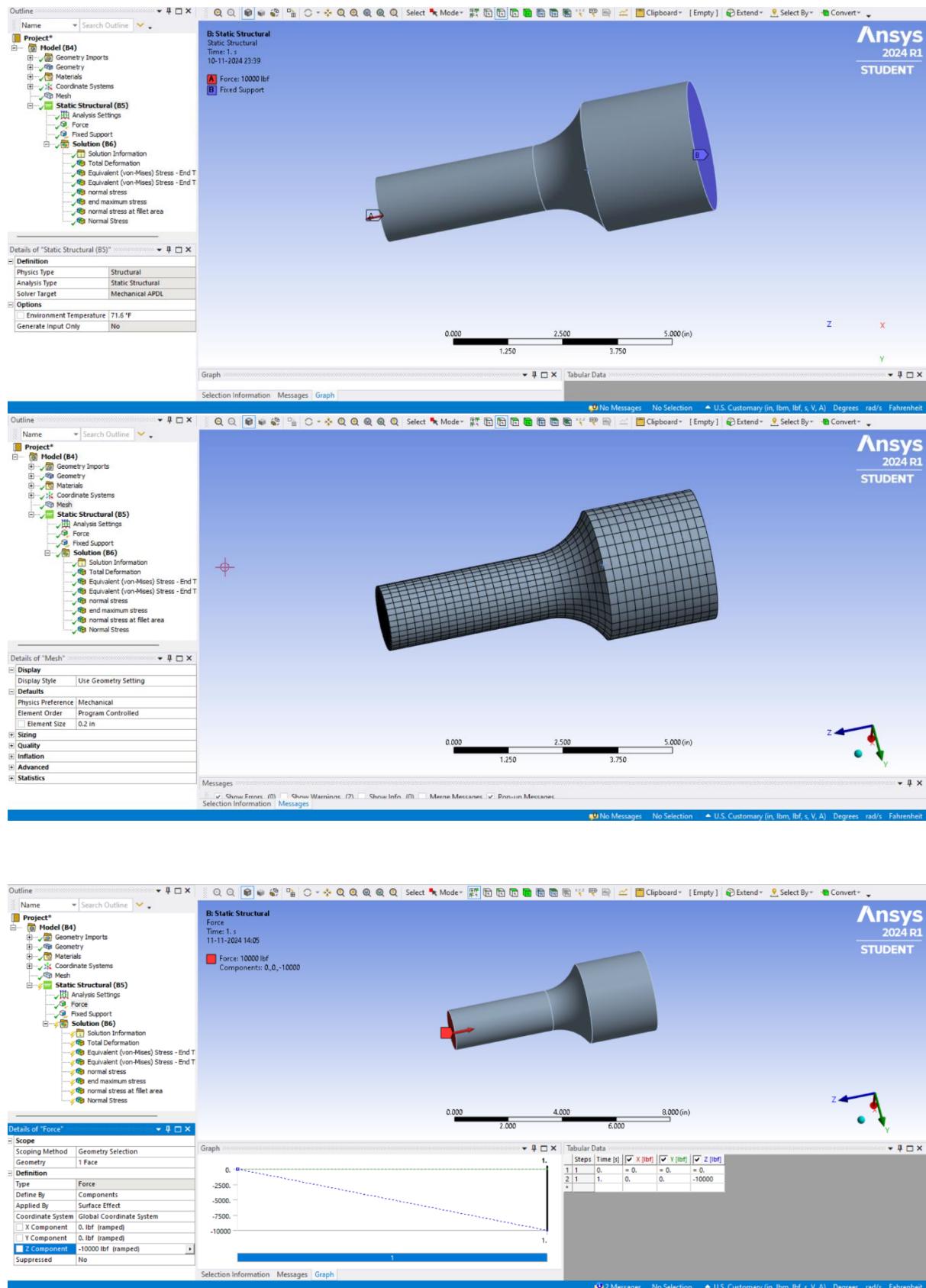
$$\sigma_{\text{max}} \Rightarrow Kt \times \sigma_{\text{nominal}}$$

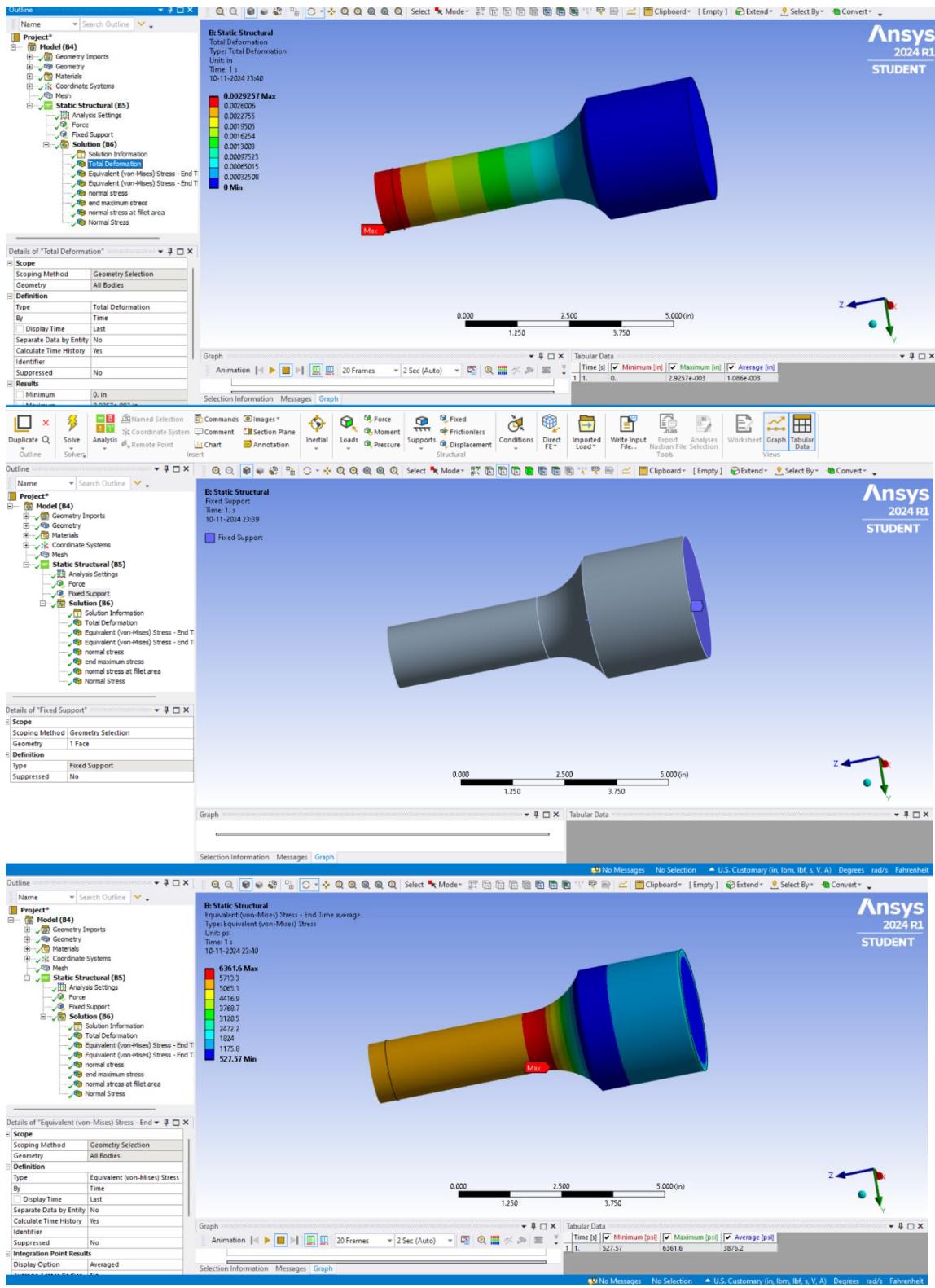
$$\Rightarrow 2.5 \times 1.724 \times 10^5 \Rightarrow 3.442 \times 10^5 \text{ psi}$$

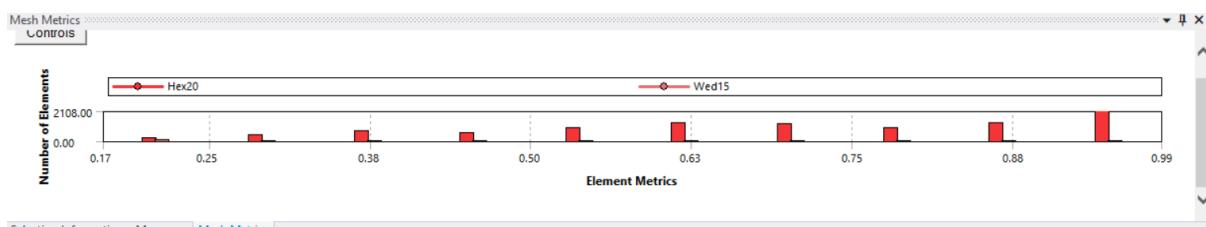
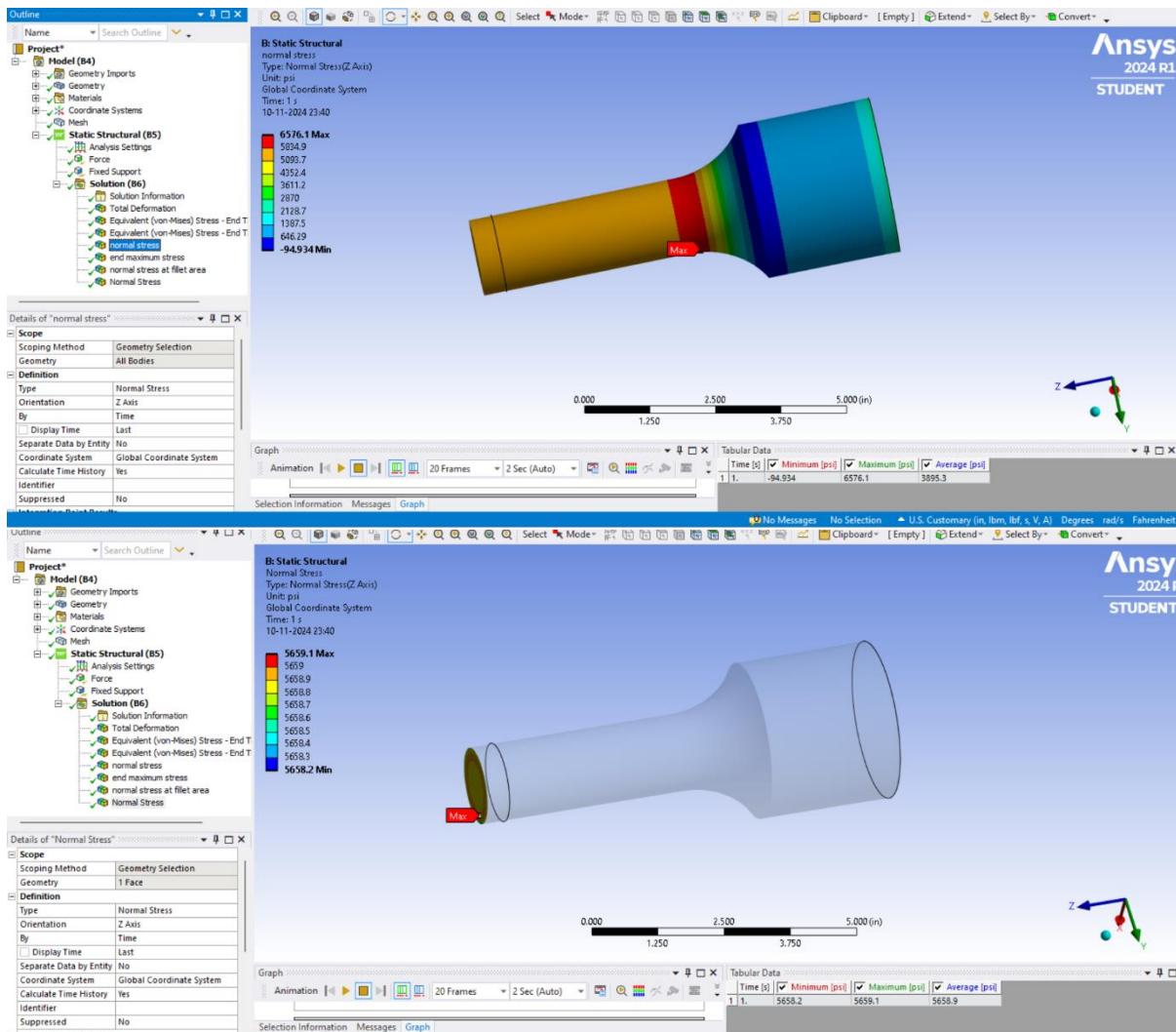
$$Y = \frac{P \times L^3}{3 \times E \times I} \Rightarrow \frac{10000 \times (11.5)^3}{3 \times 1.029 \times 10^7 \times 0.667}$$

$$\Rightarrow 0.00624972.$$

TASK3







Properties of Outline Row 4: Aluminum Alloy			
	A	B	C
1	Property	Value	Unit
2	Density	0.10007	lb in^-3
3	Isotropic Secant Coefficient of Thermal Expansion		
5	Isotropic Elasticity		
6	Derive from	Young's Modulus and Poisson's...	
7	Young's Modulus	1.0298E+07	psi
8	Poisson's Ratio	0.33	
9	Bulk Modulus	1.0096E+07	psi
10	Shear Modulus	3.8713E+06	psi
11	S-N Curve	Tabular	
15	Tensile Yield Strength	40611	psi
16	Compressive Yield Strength	40611	psi
17	Tensile Ultimate Strength	44962	psi
18	Compressive Ultimate Strength	0	psi

EXAMPLE 1

TASK-3.

$$D = 1.5 \text{ inch}$$

$$d = 0.75 \text{ inch}$$

$$1.029 \times 10^9 \text{ Young's module}$$

$$0.33 \text{ Possion's ratio}$$

$$\text{Area} = \frac{\pi d^2}{4} = \frac{\pi (0.75)^2}{4} = 0.4418 \text{ in}^2$$

nominal stress σ_{nominal}

$$\sigma_{\text{nominal}} = \frac{F}{A} = \frac{10000 \text{ lbs}}{0.4418} = 22640 \text{ psi}$$

Determine the stress concentration factor (K_t)

$$\frac{D}{d} = \frac{1.5}{0.75} = 2.$$

K_t value from table for $D/d = 2$ is

$$K_t = 2.5$$

calculating maximum stress (σ_{max}) at the fillet

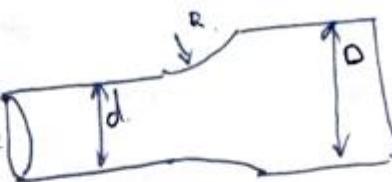
The maximum stress near the shoulder

fillet is given as

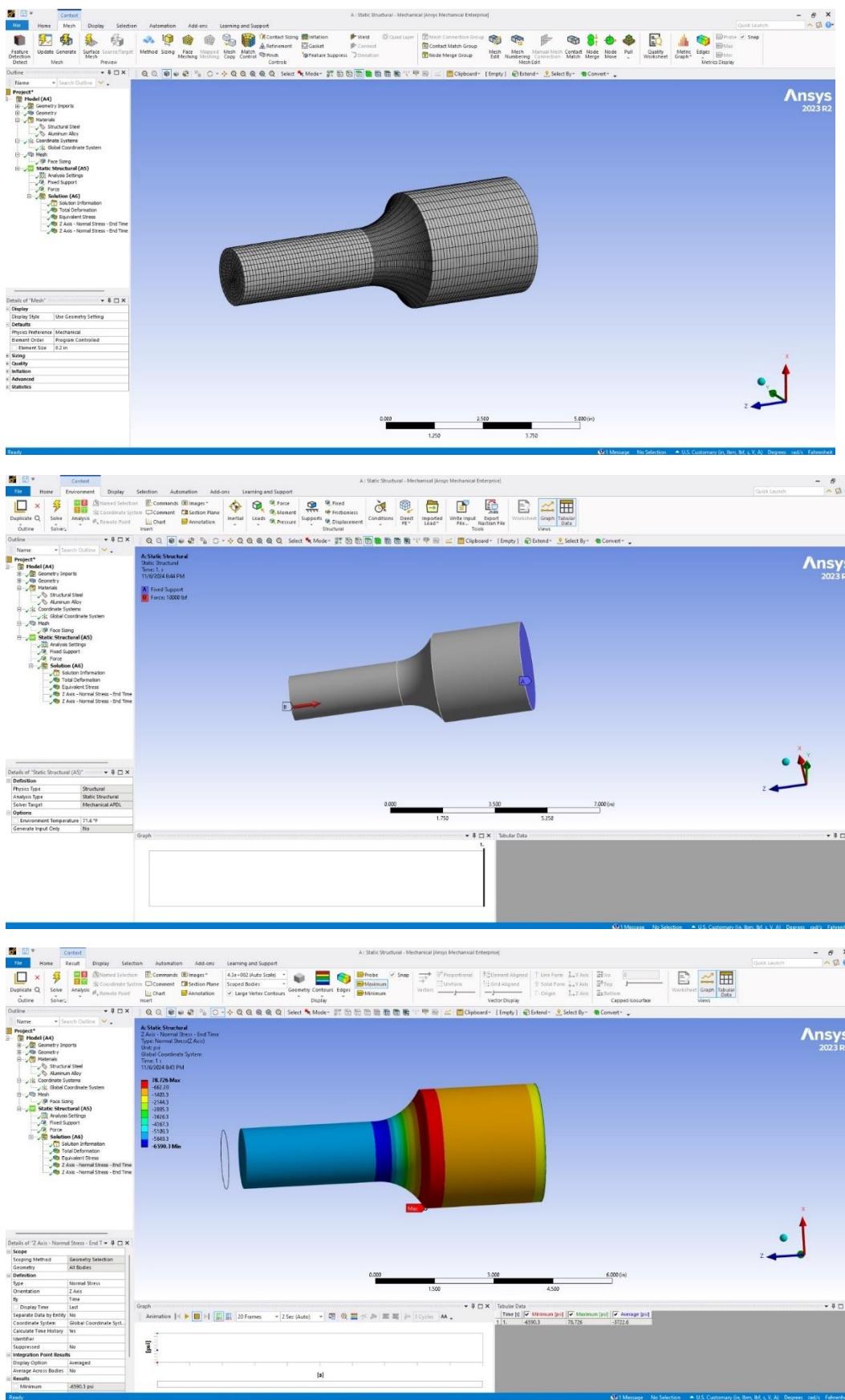
$$\sigma_{\text{max}} = K_t \times \sigma_{\text{nominal}}$$

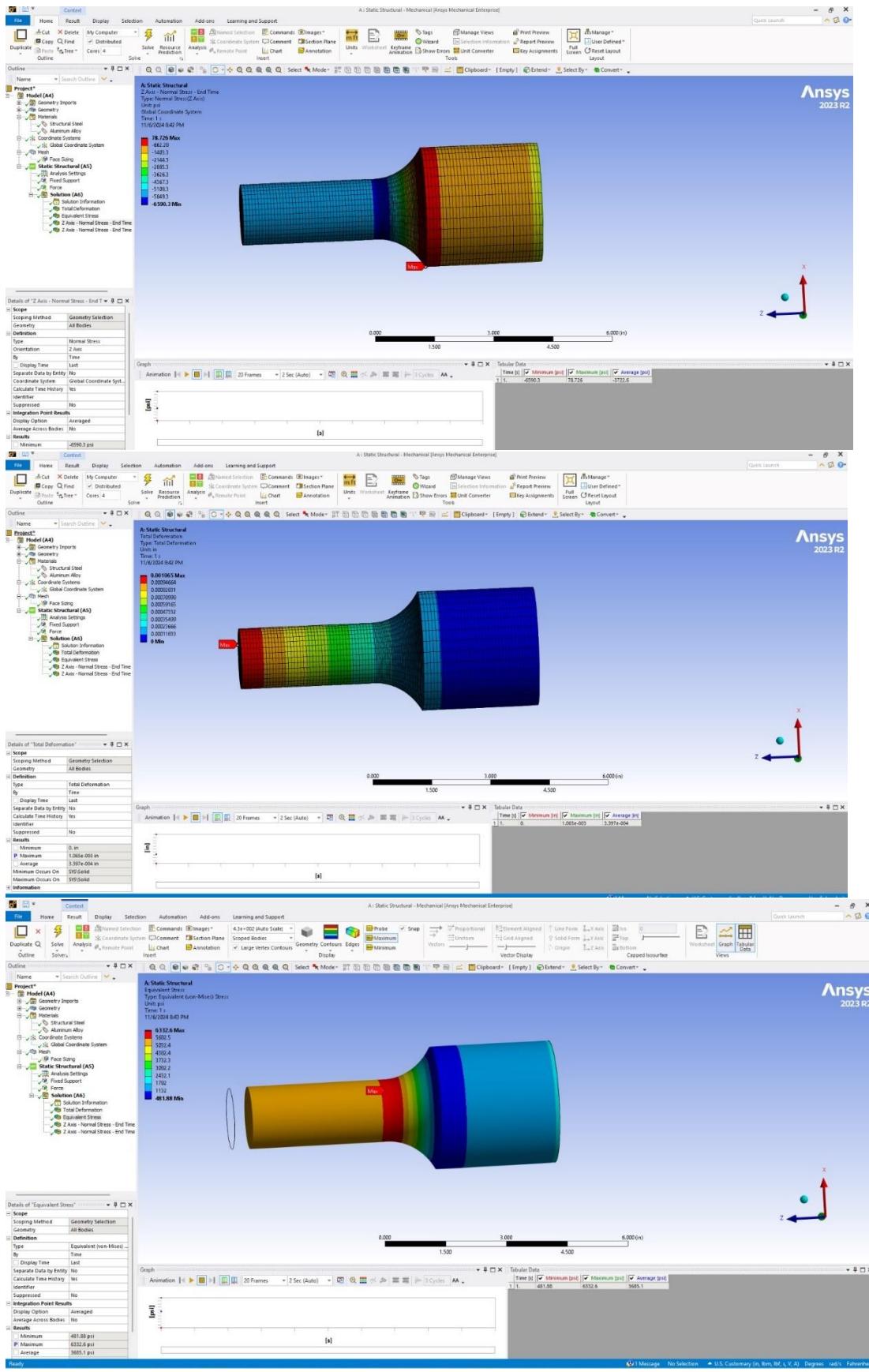
$$= 2.5 \times 22640.$$

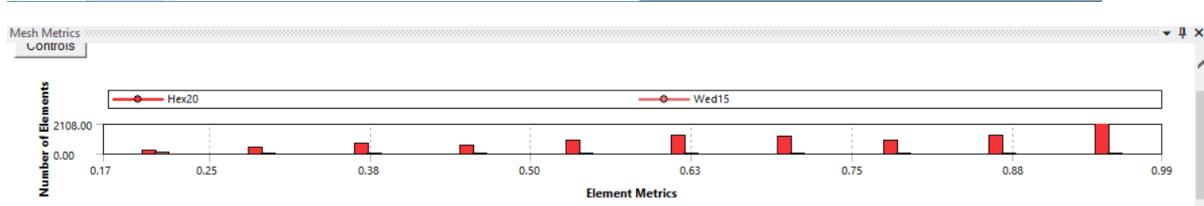
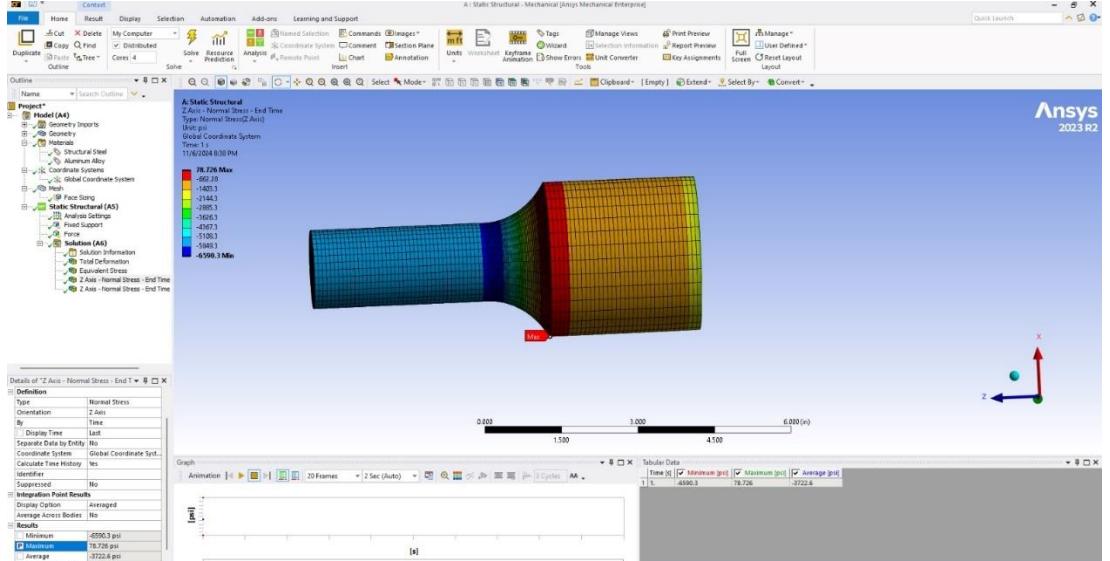
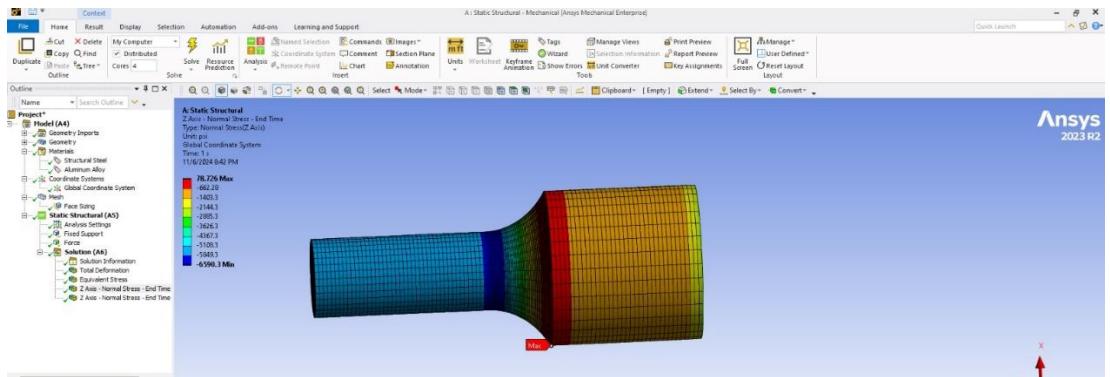
$$\Rightarrow 564100 \text{ psi}$$



EXAMPLE 2







Properties of Outline Row 4: Aluminum Alloy

	A	B	C
Property	Value	Unit	
Density	0.10007	lb in^-3	
Isotropic Secant Coefficient of Thermal Expansion			
Isotropic Elasticity			
Derive from	Young's Modulus and Poisson's...		
Young's Modulus	1.0298E+07	psi	
Poisson's Ratio	0.33		
Bulk Modulus	1.0096E+07	psi	
Shear Modulus	3.8713E+06	psi	
S-N Curve	Tabular		
Tensile Yield Strength	40611	psi	
Compressive Yield Strength	40611	psi	
Tensile Ultimate Strength	44962	psi	
Compressive Ultimate Strength	0	psi	

EXAMPLE -2

Small diameter (d) = 0.75 inches

Large diameter (D) = 2 inches

Applied load = 10000 lbs.

Material - Aluminium.

Young's modulus $E = 1029 \times 10^7$ psi

Poisson Ratio = 0.33

Cross-sectional Area (A)

$$A = \frac{\pi d^2}{4} = \frac{\pi (0.75)^2}{4} = 0.4418 \text{ in}^2$$

Nominal stress:

$$\sigma_{\text{nominal}} = \frac{F}{A} = \frac{10000 \text{ lbs}}{0.4418} = 22,640 \text{ psi}$$

Stress concentration factor.

$$\text{Diameter Ratio } \frac{D}{d} = \frac{2}{0.75} = 2.67$$

From Table. $K_f = 2.67$ the K_f value is

2.7

maximum stress $\sigma_{\text{max}} = K_f \times \sigma_{\text{nominal}}$.

$$\Rightarrow 2.7 \times 22640$$

$$= 61188 \text{ psi}$$