

8.03 Pset 2
Problem 1

Key concept: There are designed harmonic oscillators
→ damaged slugs down motion

displacement $\delta(x)$ from equilibrium at position x where $\partial\delta/\partial x = 0$.

a) Find $\frac{d^2x}{dt^2}$ & $\frac{dx}{dt}$ if displacement of pen is critically damped & damped harmonic oscillator

$\omega_0 = 0$ & $\dot{s}(t) = V_0$. Dess charge sign before equation im Rahmen der S. 50

$$\frac{-\Delta t \pm \sqrt{\delta^2 - 4W_0^2}}{2} = -\frac{1}{2} \pm \sqrt{\frac{V_0^2 - W_0^2}{4}} \Rightarrow e^{-\frac{It}{2}} \quad (\text{+/-})$$

get. $s(t) = 0$ $\frac{X}{4} = \omega_0 t \Rightarrow \omega_0 = \frac{X}{2t}$ $\gamma = 2\omega_0$

$$H^2 + \gamma H + \omega_0^2 = 0 \Rightarrow \omega_0 = \frac{\sqrt{H^2 + 4\gamma H}}{2} = \frac{\sqrt{(H+\gamma)^2 - \gamma^2}}{2} = \frac{\sqrt{(H+\gamma)^2 - (H-\gamma)^2}}{2}$$

$$\Rightarrow x(t) = A e^{-(H+\gamma)t} + B e^{(H-\gamma)t}$$

$$= \frac{(A+Be^{-\gamma t})(e^{-Ht} + Be^{-\gamma t})}{e^{-2\gamma t}} + X \omega_0 e^{-\gamma t} = \frac{Ae^{-Ht} + Ae^{-\gamma t}Be^{-\gamma t} + Be^{-Ht} + Be^{-\gamma t}Be^{-\gamma t}}{e^{-2\gamma t}} + X \omega_0 e^{-\gamma t}$$

$$= \frac{Ae^{-Ht} + Be^{-Ht} + (A+B)e^{-\gamma t}}{e^{-2\gamma t}} + X \omega_0 e^{-\gamma t} = \frac{Ae^{-Ht} + Be^{-Ht}}{e^{-2\gamma t}} + (A+B)e^{-\gamma t} + X \omega_0 e^{-\gamma t}$$

$$\Rightarrow x(t) = A e^{-Ht} + B e^{-Ht} + (A+B)e^{-\gamma t} + X \omega_0 e^{-\gamma t}$$

$$\dot{x}(t) = -Ae^{-Ht} - Be^{-Ht} - (A+B)\gamma e^{-\gamma t} - X \omega_0 \gamma e^{-\gamma t} \Rightarrow A = 0$$

$$\dot{s}(t) = V_0 e^{-Ht} + (A+B)\gamma e^{-\gamma t} - \omega_0 \gamma e^{-\gamma t} \Rightarrow B = V_0$$

$$s(t) = V_0 t + e^{-\frac{Ht}{2}} + \frac{V_0 e^{-\frac{Ht}{2}}}{\gamma}$$

Based in $A(s) = -V_0 e^{-\omega s} + V_0 e^{\omega s}$ where $s = j\omega$ (j is Imaginary unit)

$$A(s) = 0 \Rightarrow (A + Bt)e^{\omega s} = A = 0 \Rightarrow A = 0$$

$$\dot{S}(s) = V_0 e^{-\omega s} + (A + Bt)e^{\omega s} \Rightarrow B = V_0$$

$$S(t) = V_0 t e^{-\frac{\omega}{2}t} + V_0 e^{-\frac{\omega}{2}t}$$

b) Find the response of an overdamped pen so initial conditions $s(0) = s_0$ & $\dot{s}(0) = 0$

$$\begin{aligned} \text{only damped: } & \frac{t}{4} - \omega_0 t > 0 \\ \text{if: } & \Re(t) = A e^{-\sigma t} + B e^{-\sigma t} i \quad \text{where } T_1, T_2 \in \mathbb{C} \pm \sqrt{\frac{T^2}{4} - \omega_0^2} \\ \text{then: } & \begin{cases} \Re(t) = -T_1 A e^{-T_1 t} - T_2 B e^{-T_2 t} \\ \Im(t) = T_2 A e^{-T_1 t} - T_1 B e^{-T_2 t} \end{cases} \\ \text{so: } & \begin{cases} 0 = -T_1 A - T_2 B = 0 \\ 0 = T_2 A - T_1 B = 0 \end{cases} \quad \Rightarrow A = B = 0 \\ \text{so: } & \boxed{\mathcal{E}(t) = A + B t} \\ \text{and: } & A = \frac{T_1 + T_2}{2} S_0 \\ \text{so: } & \begin{cases} B = -\frac{T_1 - T_2}{2} S_0 \\ \Re(t) = \frac{T_1 + T_2}{2} S_0 e^{-\sigma t} \end{cases} \end{aligned}$$

c) For blower & condenser we consider eqn / labour position we $S=0$!
 $f(S(t)) = 0$ $A(e^{-\pi t} + \frac{T_1}{T_2} e^{-T_2 t}) \Rightarrow \frac{T_1}{T_2} e^{-T_2 t} = 1$

$$T_1 e^{-T_2 t} = T_2 e^{-T_1 t}$$

using T₁ for many classes, no one short.

displacement $\delta(x)$ from equilibrium at position x where $\partial\delta/\partial x = 0$.

a) Find $\frac{d^2x}{dt^2}$ & $\frac{dx}{dt}$ if displacement of pen is critically damped & over damped harmonic oscillator

$\omega_0 = 0$ & $\dot{s}(t) = V_0$. Dess charge sign before equation in bracket
are $s(t)$

$$\frac{-\Delta t \sqrt{V^2 - 4W_0^2}}{2} = -\frac{\Delta t}{2} \pm \sqrt{\frac{V^2}{4} - W_0^2}$$

$$\text{get } s(t) = 0 \quad \frac{V}{4} = \omega_0 t \Rightarrow \omega_0 = \frac{V}{2\Delta t} \quad \gamma = 2\omega_0 \Rightarrow e^{-\gamma t} = e^{-\frac{Vt}{\Delta t}}$$

$$s(t) = \frac{V}{4} e^{-\frac{Vt}{\Delta t}}$$

$$\frac{(H^2 + 2\omega_0^2)^{1/2} + (H^2 + 2\omega_0^2)^{1/2}}{2} = \frac{dX}{dt} (e^{i\omega_0 t}) + X \omega_0 e^{i\omega_0 t} + Y = \frac{dX}{dt} (e^{i\omega_0 t}) + X \omega_0 e^{i\omega_0 t} + \frac{dY}{dt} (e^{i\omega_0 t}) + Y \omega_0 e^{i\omega_0 t} = \frac{dX}{dt} (e^{i\omega_0 t}) + (X \omega_0 e^{i\omega_0 t} + Y \omega_0 e^{i\omega_0 t}) + \frac{dY}{dt} (e^{i\omega_0 t})$$

$$X e^{i\omega_0 t} = A + B t \Rightarrow X = (A + B t) e^{-i\omega_0 t} \Rightarrow A = 0$$

$$S(t) = 0 = (A + B t) e^{i\omega_0 t} \Rightarrow A = 0$$

$$\dot{S}(t) = V_0 = B e^{-i\omega_0 t} + (A + B t) - \omega_0 - i\omega_0 t \Rightarrow B = V_0$$

$$S = V_0 t + e^{-i\omega_0 t} - \frac{V_0 t}{2} \Rightarrow V_0 t e^{-\frac{i\omega_0 t}{2}}$$

b) Find the response of an off-damped pen to initial conditions $s(0)=x_0$ & $\dot{s}(0)=0$

$$\begin{aligned} \text{only damped: } & \frac{t}{4} - \omega_0 t > 0 \\ \text{if: } & \omega(t) = A e^{-\nu t} + B e^{-\tau t} \quad \text{where } T_1, T_2 \neq \frac{\pi}{2} \pm \sqrt{\frac{T^2}{4} - \omega_0^2} \\ \text{if: } & \dot{\omega}(t) = -T_1 A e^{-\nu t} - T_2 B e^{-\tau t} \\ & \dot{\omega}(t) = -T_1 A - T_2 B = 0 \quad T_1 A - T_2 B = 0 \quad -T_1 A = T_2 B \\ & \theta = -\frac{T_1}{T_2} A \quad B = -\frac{T_1}{T_2} A \\ & \omega(t) = A e^{-\nu t} + \left(-\frac{T_1}{T_2} A \right) e^{-\tau t} \\ & \omega(t) = \frac{A}{T_2} \left(e^{-\nu t} - T_1 e^{-\tau t} \right) \end{aligned}$$

c) For blower & condenser we consider eqn / labour position we $S=0$!
 $f(S(t)) = 0$ $A(e^{-\pi t} + \frac{T_1}{T_2} e^{-T_2 t}) \Rightarrow \frac{T_1}{T_2} e^{-T_2 t} = -A e^{-\pi t}$

$$T_1 e^{-T_2 t} = T_2 e^{-T_1 t}$$

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$$\frac{-\Delta t \pm \sqrt{\delta^2 - 4W_0^2}}{2} = -\frac{1}{2} \pm \sqrt{\frac{V_0^2}{4} - W_0^2}$$

$$\text{get. } s(t) = 0 \quad \frac{X}{4} = \omega_0 t \Rightarrow \omega_0 = \frac{X}{2t} \quad Y = 2\omega_0 t \Rightarrow e^{-\frac{Y}{2\omega_0}} = e^{-\frac{X}{\omega_0 t}}$$

$$z(t) = \frac{X}{4} e^{-\frac{Y}{2\omega_0}} = \frac{(X + 2\omega_0 t)^2 + (Y - 2\omega_0 t)^2}{16} = \frac{H^2 + 2\omega_0^2 t^2}{16}$$

$$\frac{dz(t)}{dt} = \frac{X}{4} e^{-\frac{Y}{2\omega_0}} \cdot (-\frac{Y}{2\omega_0}) = \frac{dX}{dt} e^{-\frac{Y}{2\omega_0}} + X \omega_0 e^{-\frac{Y}{2\omega_0}} + Y = \frac{dX}{dt} e^{-\frac{Y}{2\omega_0}} + X \omega_0 e^{-\frac{Y}{2\omega_0}} + X \omega_0^2 t e^{-\frac{Y}{2\omega_0}} + (Y - 2\omega_0 t) \frac{d}{dt} e^{-\frac{Y}{2\omega_0}} = \frac{dX}{dt} e^{-\frac{Y}{2\omega_0}} + (Y - 2\omega_0 t) \frac{d}{dt} e^{-\frac{Y}{2\omega_0}} + X \omega_0^2 t e^{-\frac{Y}{2\omega_0}} + (Y - 2\omega_0 t) \frac{d}{dt} e^{-\frac{Y}{2\omega_0}} = \frac{dX}{dt} e^{-\frac{Y}{2\omega_0}} + (A + Bt) e^{-\frac{Y}{2\omega_0}} = A = 0$$

$$\dot{s}(t) = V_0 e^{-\frac{Y}{2\omega_0}} + (A + Bt) e^{-\frac{Y}{2\omega_0}} \Rightarrow B = V_0$$

$$s(t) = V_0 t e^{-\frac{Y}{2\omega_0}} + \frac{1}{2} t^2 e^{-\frac{Y}{2\omega_0}}$$

b) Find the response of an off-damped pen to initial conditions $s(0)=x_0$ & $\dot{s}(0)=0$

$$\begin{aligned} \text{only damped: } & \frac{t}{4} - \omega_0 t > 0 \\ \text{if: } & \omega(t) = A e^{-\nu t} + B e^{-\tau t} \quad \text{where } T_1, T_2 \neq \frac{\pi}{2} \pm \sqrt{\frac{T^2}{4} - \omega_0^2} \\ \text{if: } & \dot{\omega}(t) = -T_1 A e^{-\nu t} - T_2 B e^{-\tau t} \\ & \dot{\omega}(t) = -T_1 A - T_2 B = 0 \quad T_1 A - T_2 B = 0 \quad -T_1 A = T_2 B \\ & \theta = -\frac{T_1}{T_2} A \quad B = -\frac{T_1}{T_2} A \\ & \omega(t) = A e^{-\nu t} + \left(-\frac{T_1}{T_2} A \right) e^{-\tau t} \\ & \omega(t) = \frac{A}{T_2} \left(e^{-\nu t} - \frac{T_1}{T_2} e^{-\tau t} \right) \end{aligned}$$

c) For blower & condenser we consider eqn / labour position we $S=0$!
 $f(S(t)) = 0$ $A(e^{-\pi t} + \frac{T_1}{T_2} e^{-T_2 t}) \Rightarrow \frac{T_1}{T_2} e^{-T_2 t} = 1$

$$T_1 e^{-T_2 t} = T_2 e^{-T_1 t}$$

Part 2

17/6/19

Question 3: Oscillating Traffic Light



a) Undamped forced vibration
 $\ddot{x} + \gamma x + \omega_0^2 x = 0$
 damped mass-spring oscillator
 $m \ddot{x} + c x + m \omega_0^2 x = 0$

Amplitude decreases over time due to damping.

b) damped vibration is in suspension system. When is the decay time of traffic light amplitude oscillation after separation?

$$Y = \frac{1}{m} \text{ because } \ddot{X} + \gamma \dot{X} + \omega_0^2 X = \frac{F_0}{m} e^{-\gamma t}$$

c) Analyze expression for $y(t)$ after separation. Assume light was stationary at prev equilibrium position at time of separation $t=0$.

$$\dot{y} + \gamma \dot{y} + \omega_0^2 y = 0$$

$$y(t) = A e^{-\gamma t} \cos(\omega t + \phi)$$

$$y(0) = A e^{0} \cos(\omega t + \phi) = 0$$

$$y'(0) = -A \gamma e^{0} \sin(\omega t + \phi) = 0$$

$$A \gamma e^{0} = 0 \Rightarrow A \gamma = 0$$

$$A = \frac{-\omega}{\gamma} e^{0} = \frac{-\omega}{\gamma} e^{-\gamma t}$$

$$y(t) = (-\omega/\gamma) e^{-\gamma t} \cos(\omega t + \phi)$$

$$y(t) = (-\omega/\gamma) e^{-\gamma t} \cos(4.288t + 2.89)$$

Problem 4: Torsional Oscillator as a Measurement Device



Hollow cylindrical rod hung by thin bellum copper wire from stationary pos. Support - Mylar film wound on rod

- air space at 500 K between layers of Mylar - air space of 0.1 mm \Rightarrow air's gas or thin bellum increases & film.

- frequency of oscillator: 130.04 Hz , $\omega = 2.54 \text{ rad/s}$

a) Find relation between torsional frequency shift $\Delta f/f_0$ & fractional change in mass of film. (fractional just proportion)

State in mass of film/m. (fractional just proportion)
 rod: $I = \frac{\pi D^4}{32} + C \frac{d\theta}{dt} + K\theta = T(t)$ Constant damping

$I = \frac{1}{2} M R^2 \quad T = -K\theta \quad I = T$
 $\frac{d\theta}{dt} = \frac{1}{2} \theta \quad \frac{d\theta}{dt} = \frac{1}{2} \omega \quad \frac{d\theta}{dt} = \frac{1}{2} \omega \quad \frac{d\theta}{dt} = \frac{1}{2} \omega$

$I = A/M^{1/2} \quad \frac{d\theta}{dt} = (\frac{1}{2}) A M^{-3/2} = (\frac{1}{2}) M^{-1} \Rightarrow \frac{d\theta}{dt} = \frac{1}{2} M^{-1}$

b) Mass & Mylar decrease mass of speed. Calculate change in the damped rod mass / kg^2 rodable film area = η mass of film = 1.67×10^{-8}

density of one layer bellum $= 10^{-10} \text{ kg/m}^2$ fraction

What tension of a single layer does this reduction correspond to? ω^2 / η reliable

that can be reduced. Fractional change in frequency of Mylar $(\Delta f/f_0)$

$\frac{d\theta}{dt} = \frac{1}{2} \omega^2 \Rightarrow 4 \times 10^{-9} \frac{d\theta}{dt} = -2 \frac{d\theta}{dt} = 8 \times 10^{-9}$

One turn of Mylar \Rightarrow mass density of Mylar $= 2 \pi r h \rho$ (Mylar goes from both sides)

Masses $= 2 (2 \pi r h)^2 \rho \leftarrow$ surface density of Mylar

mass of one atom $\frac{M}{N_A}$ atoms/two sides $\frac{dM}{M} = \frac{2 (2 \pi r h)^2 \rho}{M} \frac{d\theta}{dt} = 2 (1.67 \times 10^{-8} \times 4 \times 10^{-9}) \frac{d\theta}{dt}$

$= 3.76 \times 10^{-20} \frac{d\theta}{dt} = 8 \times 10^{-9} \frac{d\theta}{dt} \Rightarrow \frac{d\theta}{dt} = 2 \times 10^{10} \text{ rad/s}^2$

Fraction of Mylar $= \frac{2 \times 10^{10}}{10^{12}} = 2 \times 10^{-2}$

Unit reaction: $2 \left(\frac{\rho \times \text{atomic weight}}{\text{cm}^3} \right) \times \frac{\text{atoms}}{\text{cm}^2}$

Unit atomic weight: $\frac{2 \times 10^{10}}{\text{cm}^3} \times \frac{\text{atoms}}{\text{cm}^2} = \frac{dM}{M}$

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Problem 5: Addition of Harmonic Variables

- a) Linear system driven by F_1 oscillates w/ $\omega = 2.00 \text{ cm}$
- driven by another force F_2 at same frequency, amplitude is 1.414 cm
- driven by both forces at same time: $F = F_1 + F_2$, amplitude is 1.414 cm

\Rightarrow what is the phase difference between F_1 & F_2 ?

$$\ddot{x} + \gamma x + \omega_0^2 x = \frac{F_1}{m} e^{i\omega t} + \frac{F_2}{m} e^{i(\omega t + \delta)}$$

$$m\ddot{x} + \gamma m\dot{x} + m\omega_0^2 x = -\omega^2 + i\gamma\omega + \omega_0^2$$

$m\ddot{x} + i\gamma\omega x + \omega_0^2 x = \frac{F_1}{m} e^{i\omega t} + \frac{F_2}{m} e^{i(\omega t + \delta)}$

divide both sides by m (cancel m) $\Rightarrow (\omega_0^2 - \omega^2)x + i\gamma\omega x = \frac{F_1}{m} e^{i\omega t} + \frac{F_2}{m} e^{i(\omega t + \delta)}$

$$Ae^{i\delta} \Rightarrow \frac{A}{\sqrt{\omega_0^2 - \omega^2}} e^{i\delta} = \frac{F_1}{\sqrt{\omega_0^2 - \omega^2}} e^{i\omega t} + \frac{F_2}{\sqrt{\omega_0^2 - \omega^2}} e^{i(\omega t + \delta)}$$

$$\tan \delta = \frac{F_2}{F_1}$$

$$F = F_1 + F_2 \quad X = X_1 + X_2$$

$$\delta = 180^\circ - 45^\circ = 135^\circ$$

$$X = X_1 + X_2$$

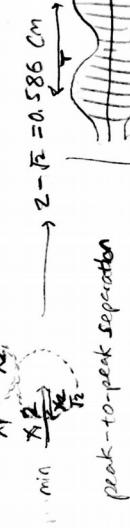
- b) Magnitude sum of two new frequencies of F_1 & F_2 are $V_1 = 100 \text{ Hz}$ & $V_2 = 102 \text{ Hz}$

Make sketch of system to sum of two forces $F = F_1 + F_2$.

Stretch amplitude vs. time after steady-state.

Give max. of min. amplitudes of peak-to-peak separation

$$max \frac{2}{\sqrt{2}} \rightarrow 2\sqrt{2} = 3.414 \text{ cm} \quad \text{amplitude vs. time after steady-state}$$



peak-to-peak separation

$$T = \frac{1}{V_1 - V_2} = \frac{1}{100 - 102} = 0.5 \text{ s}$$

Problem 6: Driven Twisted Oscillator

- torsional oscillator w/ moment of inertia I , driven by various torques between it & outer concentric ring
- restoring torque: $-K\theta$ torque due to friction: $-b\dot{\theta}$ (proportional to angular speed)
- torsion driven in steady state such that $\theta(t) = \theta_0 \cos(\omega t)$

$$a) \text{ Write down diff q. governing angular position } \theta(t) \text{ of oscillator.}$$

$$I\ddot{\theta}(t) + C\theta(t) + K\theta = -K\theta - b\dot{\theta} - \theta_0 \cos(\omega t)$$

$$I\ddot{\theta} + b\dot{\theta} + K\theta = -K\theta - b\dot{\theta} + b\dot{\theta} \quad \theta = \theta_0 \cos(\omega t)$$

$$\ddot{\theta} + \frac{b}{I}\dot{\theta} + \frac{K}{I}\theta = \frac{b}{I}\dot{\theta} \Rightarrow \theta = \theta_0 \cos(\omega t)$$

b) The angular displacement has form: $\theta(t) = \theta_0 \cos(\omega t - \phi(\omega))$

Find $\theta_0(\omega)/\dot{\theta}_0$ & make sketch of result.

$$\theta(t) = \theta_0(\omega) \cos(\omega t - \phi(\omega))$$

$$\dot{\theta}(t) = \theta_0'(\omega) \sin(\omega t - \phi(\omega))$$

$$\ddot{\theta}(t) = \theta_0''(\omega) \cos(\omega t - \phi(\omega))$$

$$\ddot{\theta} + \frac{b}{I}\dot{\theta} + \frac{K}{I}\theta = \frac{b}{I}\dot{\theta} + \frac{K}{I}\theta = \frac{b}{I}\dot{\theta} + \frac{K}{I}\theta_0 \cos(\omega t - \phi(\omega))$$

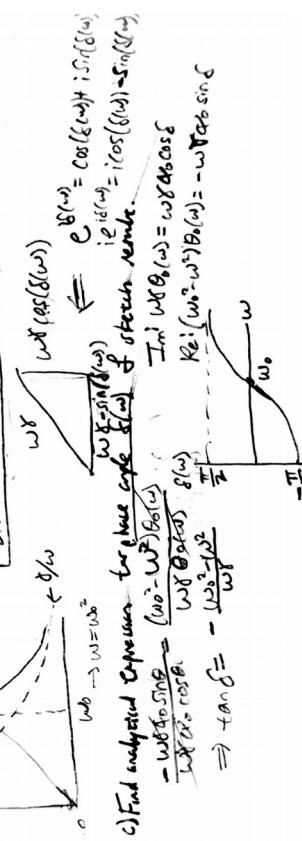
$$I\ddot{\theta} + b\dot{\theta} + K\theta = -\omega^2 + \left(\frac{K}{I}\omega + \frac{b}{I}\right)\omega^2$$

$$I\omega^2 + b\omega + K\theta_0 \cos(\omega t - \phi(\omega)) = I\omega^2 \cos(\omega t - \phi(\omega))$$

$$\Rightarrow (-\omega^2 + i\omega + \omega_0^2)\theta_0(\omega) = i\omega\theta_0'(\omega) \cos(\omega t - \phi(\omega))$$

$$\frac{\theta_0(\omega)}{\theta_0} = \frac{i\omega\theta_0'(\omega)}{(-\omega^2 + i\omega + \omega_0^2)} = \frac{i\omega\theta_0'(\omega)}{\omega_0^2 - \omega^2 + i\omega\omega_0}$$

$$\frac{\theta_0(\omega)}{\theta_0} = \frac{i\omega}{\omega_0^2 - \omega^2 + i\omega\omega_0}$$



c) Find analytical expression for base angle $\phi(\omega)$ of steady state.

$$\tan \phi = \frac{\omega_0^2 - \omega^2}{\omega^2 + \omega_0^2}$$

$$\tan \phi = -\frac{\omega_0^2 - \omega^2}{\omega^2 + \omega_0^2}$$

$$\Rightarrow \tan \phi = -\frac{\omega_0^2 - \omega^2}{\omega^2 + \omega_0^2} \sin \omega$$

