

인공지능과 설계: 해석 예측에서 설계 최적화까지

위상최적설계 (Topology Optimization) 실습

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Google Colab environment setting

Topology optimization using Python

■ Introduction

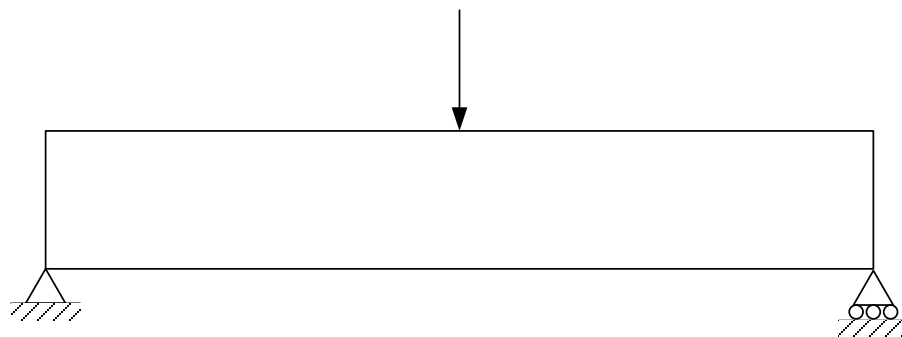
- The Python code for topology optimization problem is an open-source alternative to the 99 and 88 line MATLAB codes*.
 - Original code is provided by Topology Optimization group in DTU.
 - You can access various topology optimization apps/software in <https://www.topopt.mek.dtu.dk>.
- This code is useful for students and newcomers to the field of topology optimization in the educational sense.
 - It can be extended to include more functions such as multiple load-cases, alternative mesh-independency schemes, passive areas, etc.

*O. Sigmund, "A 99 line topology optimization code written in Matlab", *Structural Multidisciplinary Optimization* 21, 120-127, 2001.

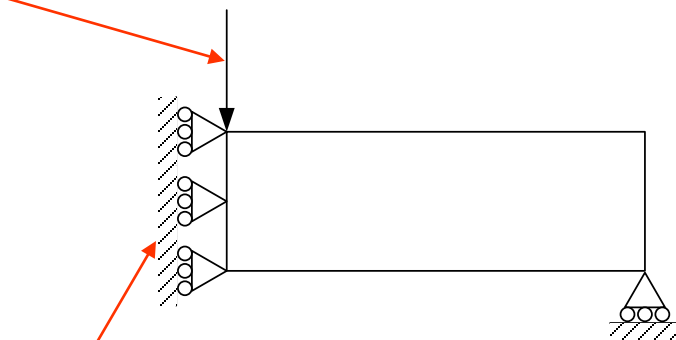
*E. Andreassen, A. Clausen, M. Schevenels, B. S. Lazarov, O. Sigmund, "Efficient topology optimization in MATLAB using 88 lines of code". *Structural Multidisciplinary Optimization* 43, 1–16, 2011.

■ Problem formulation

- The default problem in the code is **MBB-beam**.
- The modified SIMP method has various scalability advantages. (e.g. additional filters)



External load



Symmetric boundary conditions

Horizontal support

Modified SIMP; $E_{\min} = 10^{-9}$ in this practice

$$E_e(x_e) = E_{\min} + x_e^p(E_0 - E_{\min}), \quad x_e \in [0, 1]$$

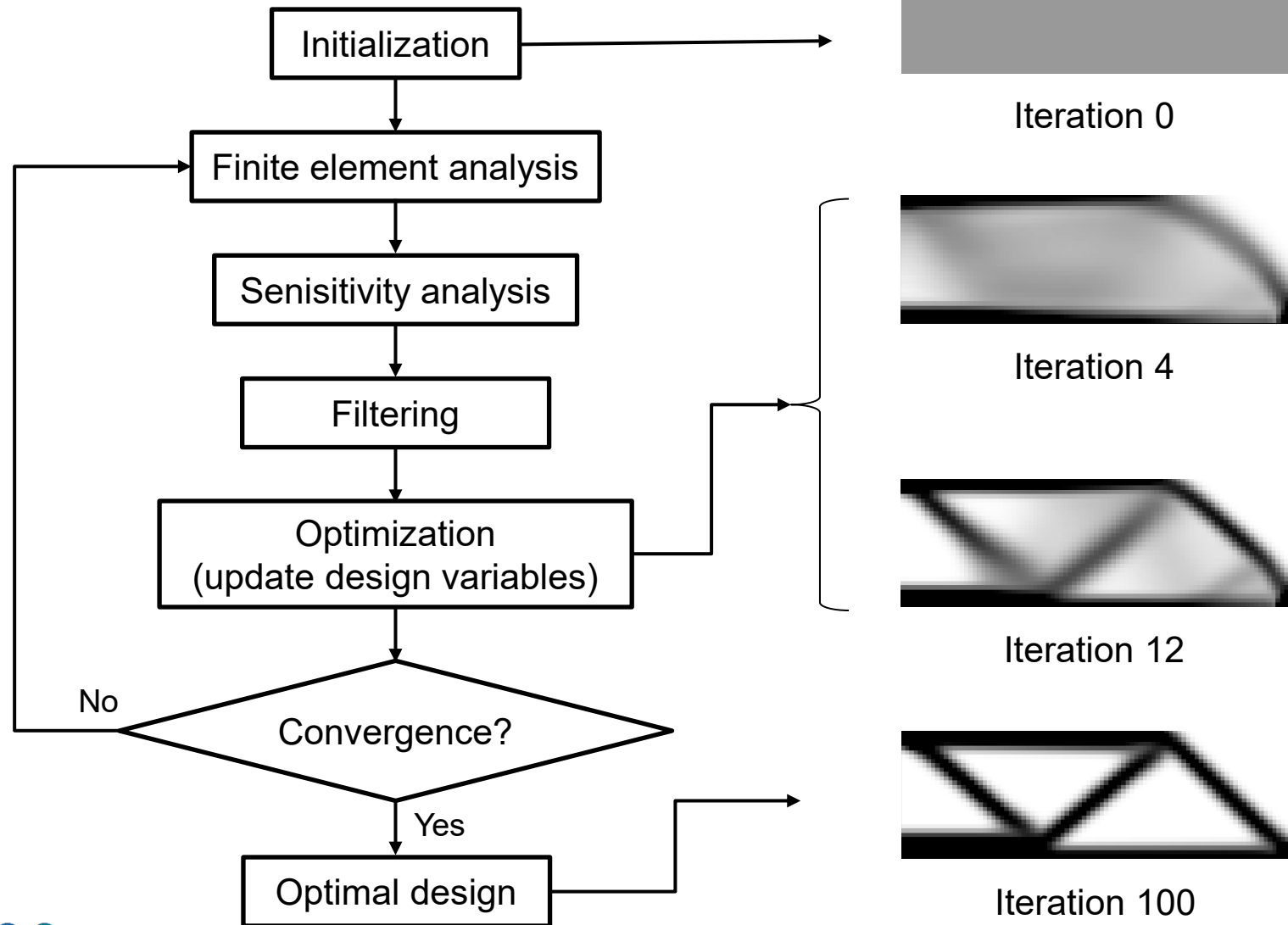
$$\min_{\mathbf{x}} : c(\mathbf{x}) = \mathbf{U}^T \mathbf{K} \mathbf{U} = \sum_{e=1}^N E_e(x_e) \mathbf{u}_e^T \mathbf{k}_0 \mathbf{u}_e$$

$$\text{subject to: } V(\mathbf{x})/V_0 = f$$

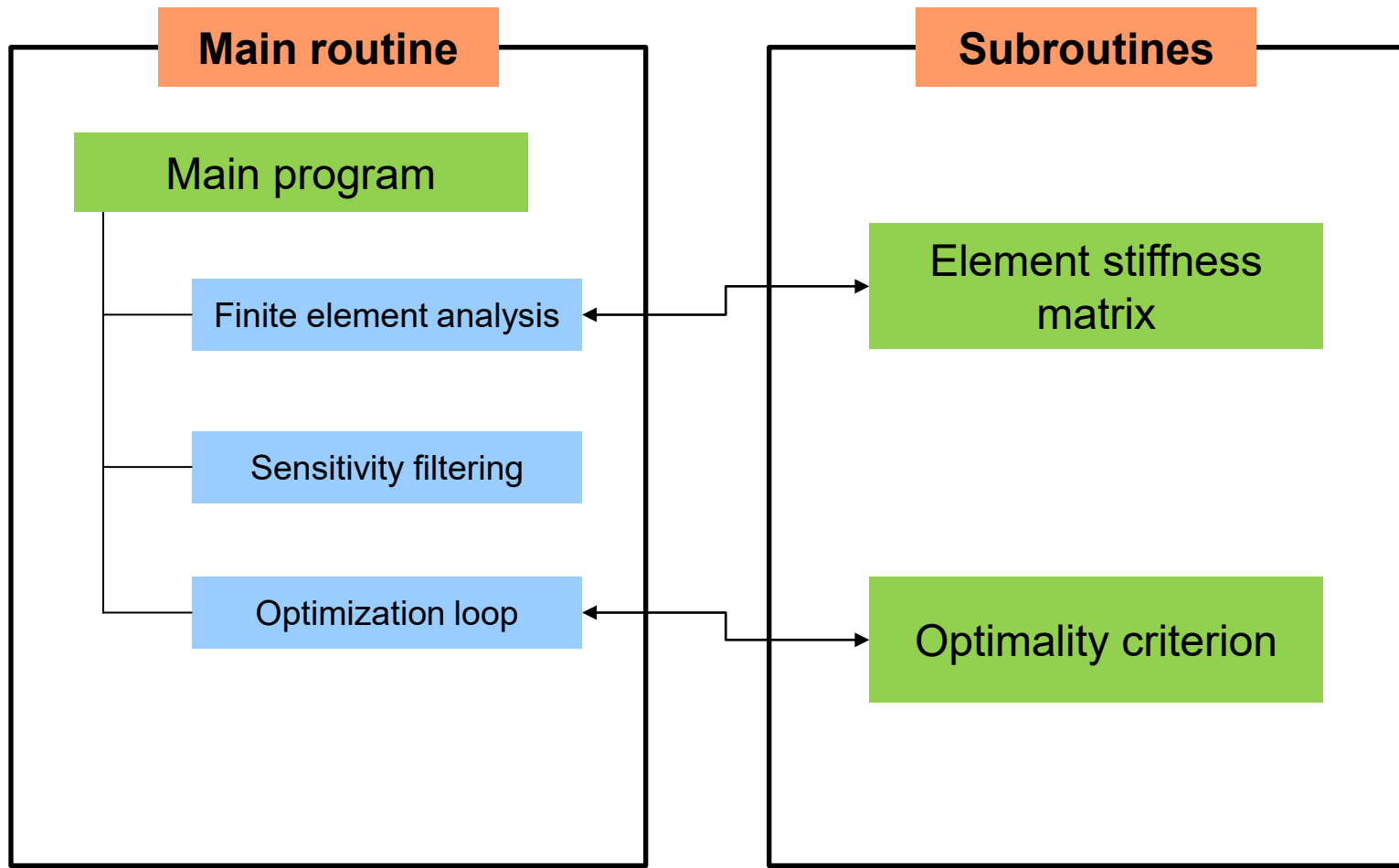
$$\mathbf{K} \mathbf{U} = \mathbf{F}$$

$$0 \leq \mathbf{x} \leq 1$$

■ Structure of the code



■ Structure of the code

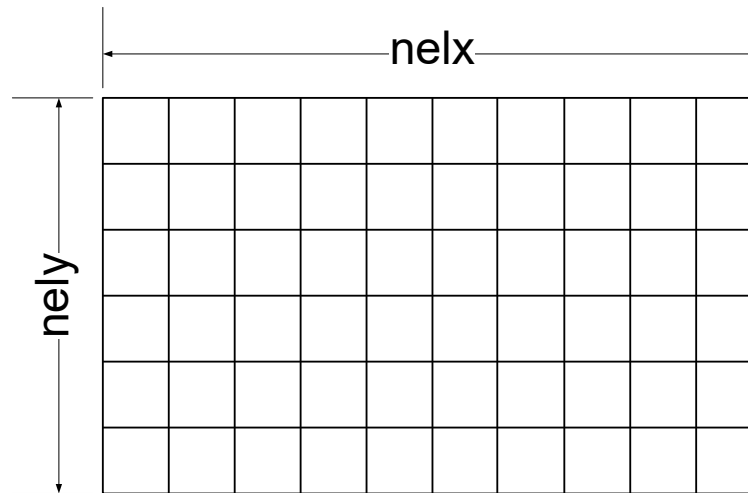


▪ Python implementation

- The main program is called from the Python command by `main(nelx,nely,volfrac,penal,rmin,ft)`

nelx: number of elements in the horizontal direction

nely: number of elements in the vertical direction



▪ Python implementation

volfrac: volume fraction ratio for a constraint

$$\text{volfrac} = \frac{\sum_{e=1}^N x_e v_e}{V_0}$$

penal: penalization power p in SIMP method

$$E_e(x_e) = E_{\min} + x_e^p (E_0 - E_{\min}), \quad x_e \in [0,1]$$

▪ Python implementation

rmin: filter radius (divided by element size)

$$\frac{\widehat{\partial c}}{\partial x_e} = \frac{1}{\max(\gamma, x_e) \sum_{i \in N_e} H_{ei}} \sum_{i \in N_e} H_{ei} x_i \frac{\partial c}{\partial x_i}$$

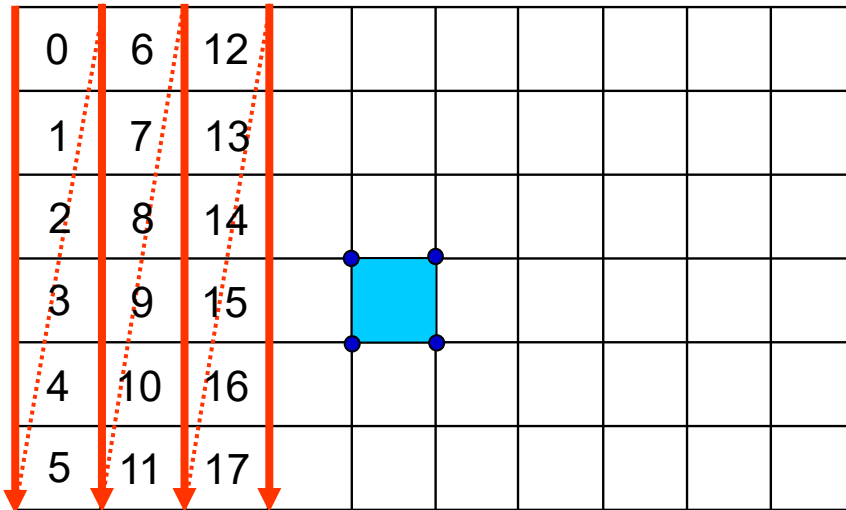
$$\text{where } \widehat{H}_{if} = \max(0, \boxed{r_{\min}} - \text{dist}(e, i))$$

ft: sensitivity filtering (**ft** = 1) or density filtering (**ft** = 2)

only use sensitivity filtering in this practice!

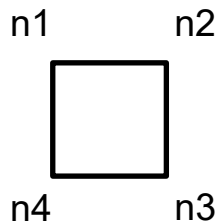
Python implementation

Element, node, and nodal displacement numbering



Local number

Global number



$$n1 = (nely+1) \times elx + ely$$

$$n2 = (nely+1) \times (elx+1) + ely$$

In this case, nelx=10, nely=6, elx=4, ely=3

$$\therefore n1=31, n2=38, n3=39, n4=32$$

Compliance and its sensitivity

$$c = \mathbf{U}^T \mathbf{F} = \mathbf{U}^T \mathbf{K} \mathbf{U} = \sum_{e=1}^N E_e(x_e) \mathbf{u}_e^T \mathbf{k}_0 \mathbf{u}_e$$

$$\frac{\partial c}{\partial \rho_i} = -\mathbf{U}^T \frac{\partial \mathbf{K}}{\partial \rho_i} \mathbf{U}$$

nodal displacement vector

$$\begin{aligned} \mathbf{u}_e &= [u_{n1,x}; u_{n1,y}; u_{n2,x}; u_{n2,y}; u_{n3,x}; u_{n3,y}; u_{n4,x}; u_{n4,y}] \\ &= [2 * n1; 2 * n1 + 1; 2 * n2; 2 * n2 + 1; \\ &\quad \dots 2 * n2 + 2; 2 * n2 + 3; 2 * n1 + 2; 2 * n1 + 3] \end{aligned}$$

2 DOFs per node, 8 DOFs per element

▪ Python implementation: Finite element analysis

FEA procedure

Step 1: Obtain stiffness matrix for each element

Step 2: Expand element stiffness matrices for assembly

Step 3: Assemble element stiffness matrices

Step 4: Build global $\mathbf{KU}=\mathbf{F}$

Step 5: Apply boundary conditions

Step 6: Determine displacement at nodes

Step 7: Determine compliance

▪ Python implementation: Finite element analysis

FEA procedure

Step 1: Obtain stiffness matrix for each element

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Step 4: Build global $\mathbf{KU}=\mathbf{F}$

Step 5: Apply boundary conditions

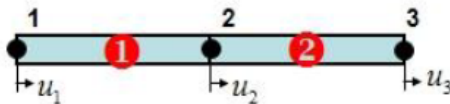
Step 6: Determine displacement at nodes

Step 7: Determine compliance

Python implementation: Finite element analysis

Stiffness Matrix Assembly

- ▶ Two bar elements (three nodes → a total of 3DOF)



If the entire structure has n DOF, expand each $[k^E]$ to an $n \times n$ matrix.

Element 1 (with nodes 1 and 2): $\begin{bmatrix} a & b \\ c & d \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix} = \begin{Bmatrix} \alpha \\ \beta \end{Bmatrix} \Rightarrow \begin{bmatrix} a & b & 0 \\ c & d & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} \alpha \\ \beta \\ 0 \end{Bmatrix}$

Element 2 (with nodes 2 and 3): $\begin{bmatrix} e & f \\ g & h \end{bmatrix} \begin{Bmatrix} u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} \beta \\ \gamma \end{Bmatrix} \Rightarrow \begin{bmatrix} 0 & 0 & 0 \\ 0 & e & f \\ 0 & g & h \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ \beta \\ \gamma \end{Bmatrix}$

$$\therefore [K] = \sum_i [K_i^E] = \begin{bmatrix} a & b & 0 \\ c & d+e & f \\ 0 & g & h \end{bmatrix}$$

■ Python implementation: Finite element analysis

```

18 # dofs:
19 ndof = 2*(nelx+1)*(nely+1)
20 # Allocate design variables (as array), initialize and allocate sens.
21 x=volfrac * np.ones(nely*nelx,dtype=float)
22 xold=x.copy()
23 xPhys=x.copy()
24 g=0 # must be initialized to use the NGuyen/Paulino OC approach
25 dc=np.zeros((nely,nelx), dtype=float)
26 # FE: Build the index vectors for the for coo matrix format.
27 KE=1k()
28 edofMat=np.zeros((nelx*nely,8),dtype=int)
29 for elx in range(nelx):
30     for ely in range(nely):
31         el = ely+elx*nely
32         n1=(nely+1)*elx+ely
33         n2=(nely+1)*(elx+1)+ely
34         edofMat[el,:]=np.array([2*n1+2, 2*n1+3, 2*n2+2, 2*n2+3,2*n2, 2*n2+1, 2*n1, 2*n1+1])
35 # Construct the index pointers for the coo format
36 iK = np.kron(edofMat,np.ones((8,1))).flatten()
37 jK = np.kron(edofMat,np.ones((1,8))).flatten()

```

2 DOFs per node

Call element (local) stiffness matrix subroutine

```

62 # BC's and support
63 dofs=np.arange(2*(nelx+1)*(nely+1))
64 fixed=np.union1d(dofs[0:2*(nely+1):2],np.array([2*(nelx+1)*(nely+1)-1]))
65 free=np.setdiff1d(dofs,fixed)
66 # Solution and RHS vectors
67 f=np.zeros((ndof,1))
68 u=np.zeros((ndof,1))
69 # Set load
70 f[1,0]=-1

```

(Symmetric boundary condition)
+ (horizontal support)

Vertical force at node 1

Python implementation: Finite element analysis

```

85 # Setup and solve FE problem
86 sK=((KE.flatten()[np.newaxis]).T*(Emin+(xPhys)**penal*(Emax-Emin))).flatten(order='F')
87 K = coo_matrix((sK,(iK,jK)),shape=(ndof,ndof)).tocsc()
88 # Remove constrained dofs from matrix
89 K = K[free,:][:,free]
90 # Solve system
91 u[free,0]=spsolve(K,f[free,0])
92 # Objective and sensitivity
93 ce[:] = (np.dot(u[edofMat].reshape(nelx*nely,8),KE) * u[edofMat].reshape(nelx*nely,8)).sum(1)
94 obj=( (Emin+xPhys**penal*(Emax-Emin))*ce ).sum()
95 dc[:]=(-penal*xPhys**(penal-1)*(Emax-Emin))*ce
96 dv[:] = np.ones(nely*nelx)

```

← Global stiffness matrix \mathbf{K}

← Solve $\mathbf{KU}=\mathbf{F}$

$$c = \mathbf{U}^T \mathbf{F} = \mathbf{U}^T \mathbf{K} \mathbf{U} = \sum_{e=1}^N E_e(x_e) \mathbf{u}_e^T \mathbf{k}_0 \mathbf{u}_e$$

$$\frac{\partial c}{\partial \rho_i} = -\mathbf{U}^T \frac{\partial \mathbf{K}}{\partial \rho_i} \mathbf{U}$$

```

119 #element stiffness matrix
120 def lk():
121     E=1
122     nu=0.3
123     k=np.array([1/2-nu/6,1/8+nu/8,-1/4-nu/12,-1/8+3*nu/8,-1/4+nu/12,-1/8-nu/8,nu/6,1/8-3*nu/8])
124     KE = E/(1-nu**2)*np.array([ [k[0], k[1], k[2], k[3], k[4], k[5], k[6], k[7]],
125     [k[1], k[0], k[7], k[6], k[5], k[4], k[3], k[2]],
126     [k[2], k[7], k[0], k[5], k[6], k[3], k[4], k[1]],
127     [k[3], k[6], k[5], k[0], k[7], k[2], k[1], k[4]],
128     [k[4], k[5], k[6], k[7], k[0], k[1], k[2], k[3]],
129     [k[5], k[4], k[3], k[2], k[1], k[0], k[7], k[6]],
130     [k[6], k[3], k[4], k[1], k[2], k[7], k[0], k[5]],
131     [k[7], k[2], k[1], k[4], k[3], k[6], k[5], k[0]] ]);
132     return (KE)

```

← Element stiffness matrix

Square bilinear 4-node element

Python implementation: Sensitivity filtering

```

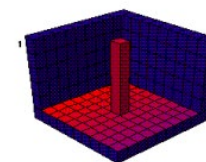
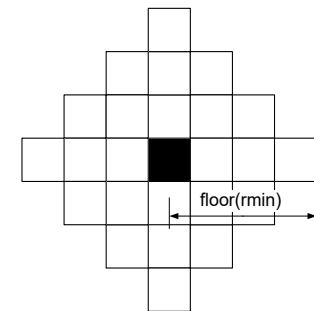
38 # Filter: Build (and assemble) the index+data vectors for the coo matrix format
39 nfilter=int(nelx*nely*((2*(np.ceil(rmin)-1)+1)**2))
40 iH = np.zeros(nfilter)
41 jH = np.zeros(nfilter)
42 sH = np.zeros(nfilter)
43 cc=0
44 for i in range(nelx):
45     for j in range(nely):
46         row=i*nely+j
47         kk1=int(np.maximum(i-(np.ceil(rmin)-1),0))
48         kk2=int(np.minimum(i+np.ceil(rmin),nelx))
49         ll1=int(np.maximum(j-(np.ceil(rmin)-1),0))
50         ll2=int(np.minimum(j+np.ceil(rmin),nely))
51         for k in range(kk1,kk2):
52             for l in range(ll1,ll2):
53                 col=k*nely+l
54                 fac=rmin-np.sqrt(((i-k)*(i-k)+(j-l)*(j-l)))
55                 iH[cc]=row
56                 jH[cc]=col
57                 sH[cc]=np.maximum(0.0, fac)
58                 cc=cc+1

97 # Sensitivity filtering:
98 if ft==0:
99     dc[:,] = np.asarray((H*(x+dc))[np.newaxis].T/Hs[:,0] / np.maximum(0.001,x))

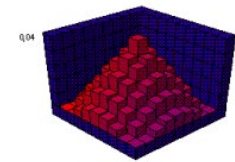
```

$$\frac{\widehat{\partial c}}{\partial x_e} = \frac{1}{\max(\gamma, x_e) \sum_{i \in N_e} H_{ei}} \sum_{i \in N_e} H_{ei} x_i \frac{\partial c}{\partial x_i}$$

where $\widehat{H}_{ei} = \max(0, r_{\min} - \text{dist}(e, i))$
 $\gamma = 10^{-3}$, $N_e = \{i \in N \mid \text{dist}(e, i) \leq r_{\min}\}$



No filtering



Filtering

Python implementation: Optimality criteria

```

103 # Optimality criteria
104 xold[:]=x
105 (x[:],g)=oc(nelx,nely,x,volfrac,dc,dv,g)

```

Call optimality criterion subroutine

```

133 # Optimality criterion
134 def oc(nelx,nely,x,volfrac,dc,dv,g):
135     l1=0
136     l2=1e9
137     move=0.2
138     # reshape to perform vector operations
139     xnew=np.zeros(nelx*nely)
140     while (l2-l1)/(l1+l2)>1e-3:
141         lmid=0.5*(l2+l1)
142         xnew[:]= np.maximum(0.0,np.minimum(x-move,np.minimum(1.0,np.minimum(x+move,x*np.sqrt(-dc/dv/lmid)))))
143         gt=g+np.sum((dv*(xnew-x)))
144         if gt>0 :
145             l1=lmid
146         else:
147             l2=lmid
148     return (xnew,gt)

```

$x_e^{\text{new}} =$

$$\begin{cases} \max(x_{\min}, x_e - m) & \text{if } x_e B_e^\eta \leq \max(x_{\min}, x_e - m), \\ x_e B_e^\eta & \text{if } \max(x_{\min}, x_e - m) < x_e B_e^\eta < \min(1, x_e + m), \\ \min(1, x_e + m) & \text{if } \min(1, x_e + m) \leq x_e B_e^\eta, \end{cases}$$

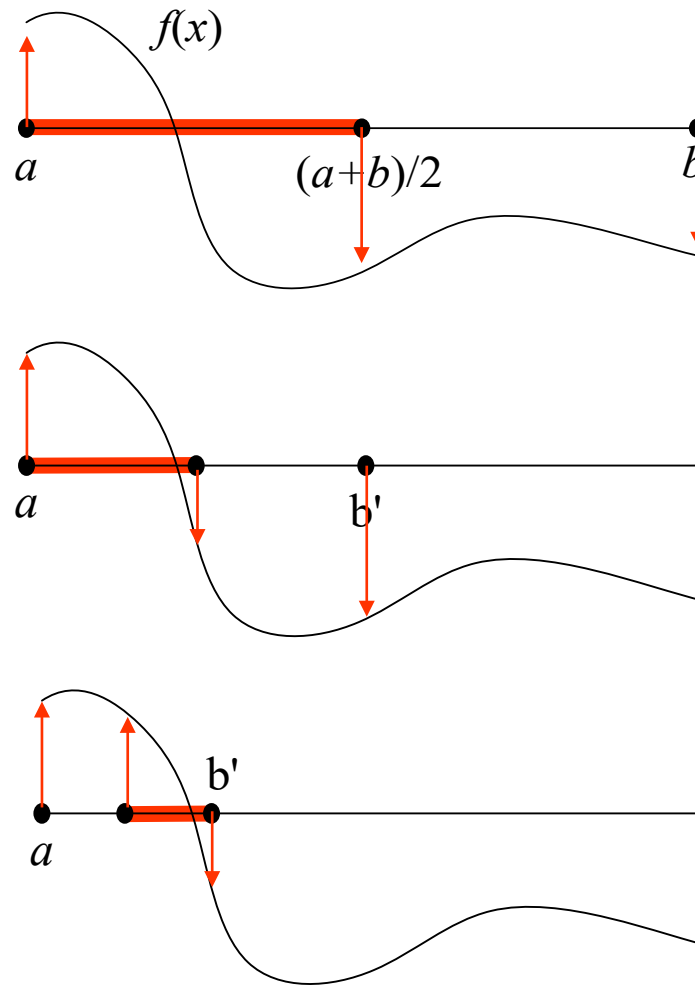
$$\eta = \frac{1}{2}; m = 0.2;$$

Lagrange multiplier λ must be chosen so that volume constraint is satisfied; the appropriate value can be found by **bisection algorithm**

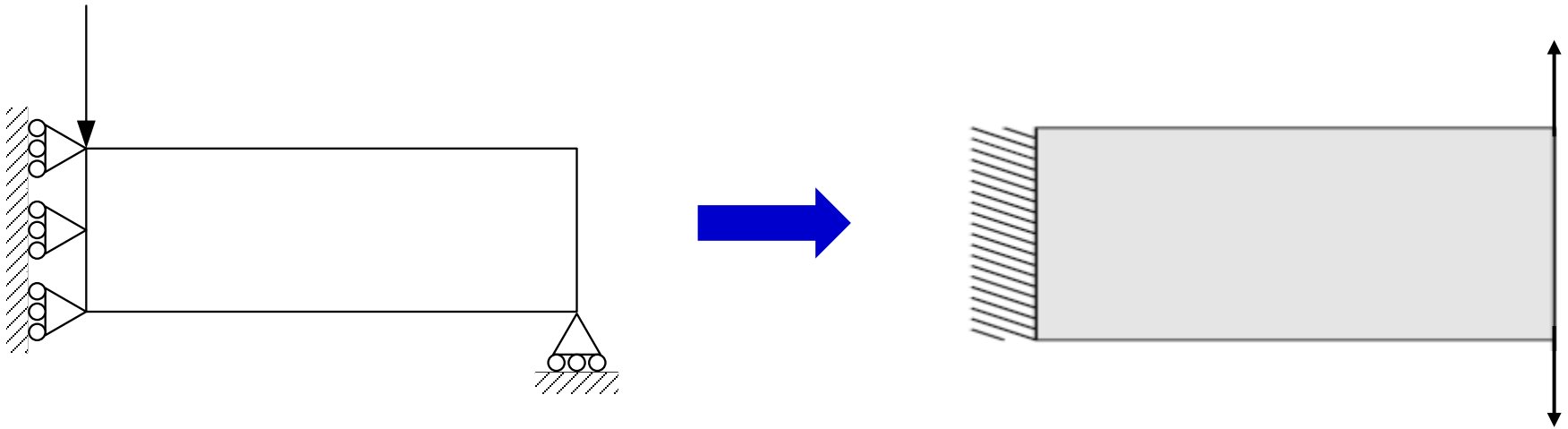
$$B_e = \frac{-\frac{\partial c}{\partial x_e}}{\lambda \frac{\partial V}{\partial x_e}}$$

Python implementation: Optimality criteria

Bisection algorithm



- **Extensions: Other boundary conditions and multiple load cases**



▪ Extensions: Other boundary conditions and multiple load cases

Other boundary conditions

```
64 fixed=np.union1d(dofs[0:2*(nely+1):2],np.array([2*(nelx+1)*(nely+1)-1]))
```



```
64 fixed=dofs[0:2*(nely+1)]
```

Multiple load cases

The objective function is now the sum of each load case,

$$c_{multiple} = \sum_{i=1}^m \mathbf{U}_i^T \mathbf{K} \mathbf{U}_i \quad \text{where } m \text{ means the number of load cases.}$$

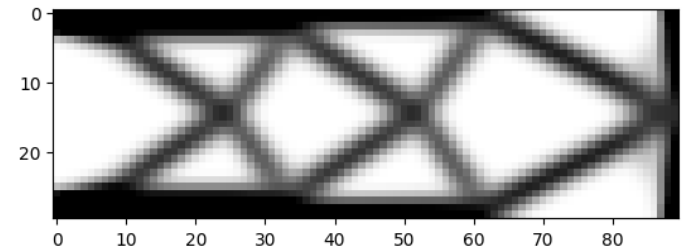
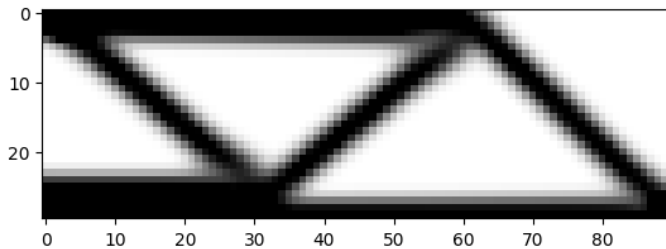
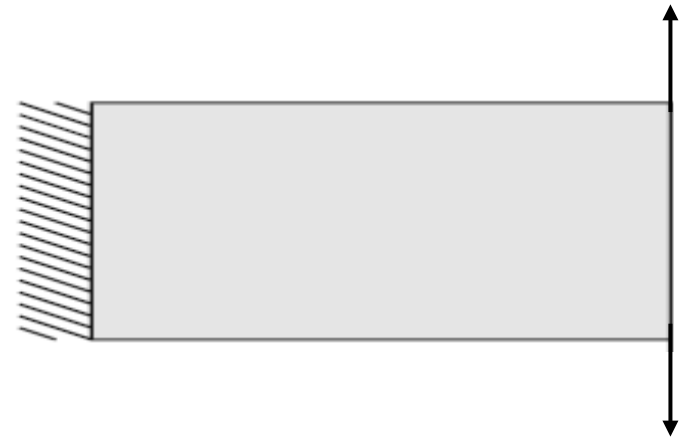
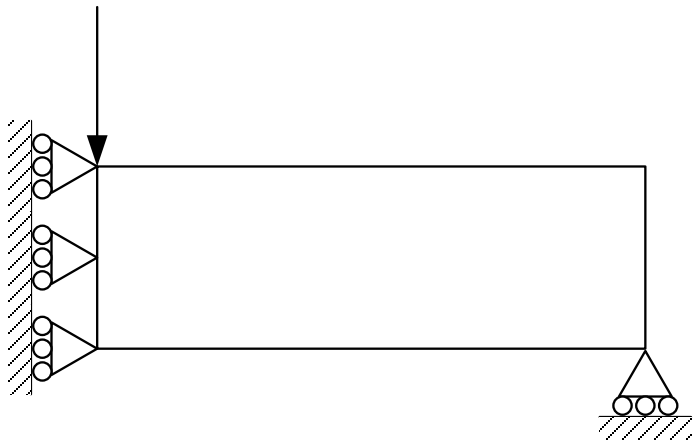
First, declare force and displacement vectors to multi-column vectors and assign load values at target index.

```
67 f=np.zeros((ndof,2))      70 f[2*(nelx)*(nely+1)+1,0]=1
68 u=np.zeros((ndof,2))      71 f[2*(nelx+1)*(nely+1)-1,1]=-1
```

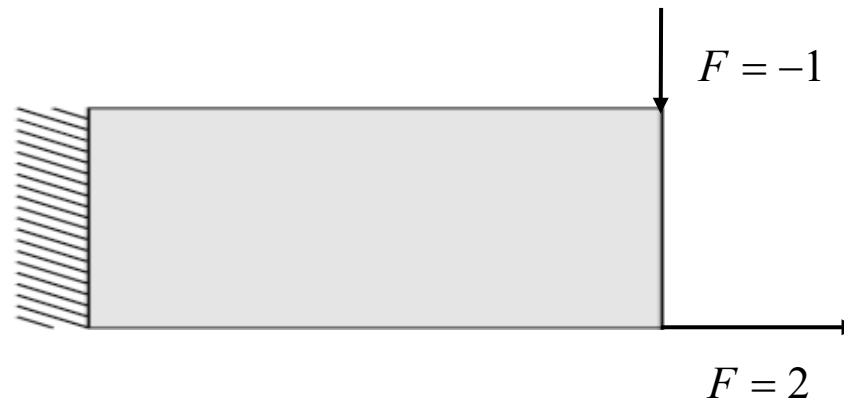
And then, change objective function and sensitivity values.

```
94 obj = 0
95 dc = np.zeros(nely*nelx)
96 for i in range(np.size(f,1)):
97     ui = u[:,i]
98     ce[:] = (np.dot(ui[edofMat].reshape(nelx*nely,8),KE) * ui[edofMat].reshape(nelx*nely,8)).sum(1)
99     obj = obj + ( (Emin*xPhys**penal*(Emax-Emin))*ce ).sum()
100    dc[:] = dc + (-penal*xPhys**(penal-1)*(Emax-Emin))*ce
```

- Extensions: Other boundary conditions and multiple load cases

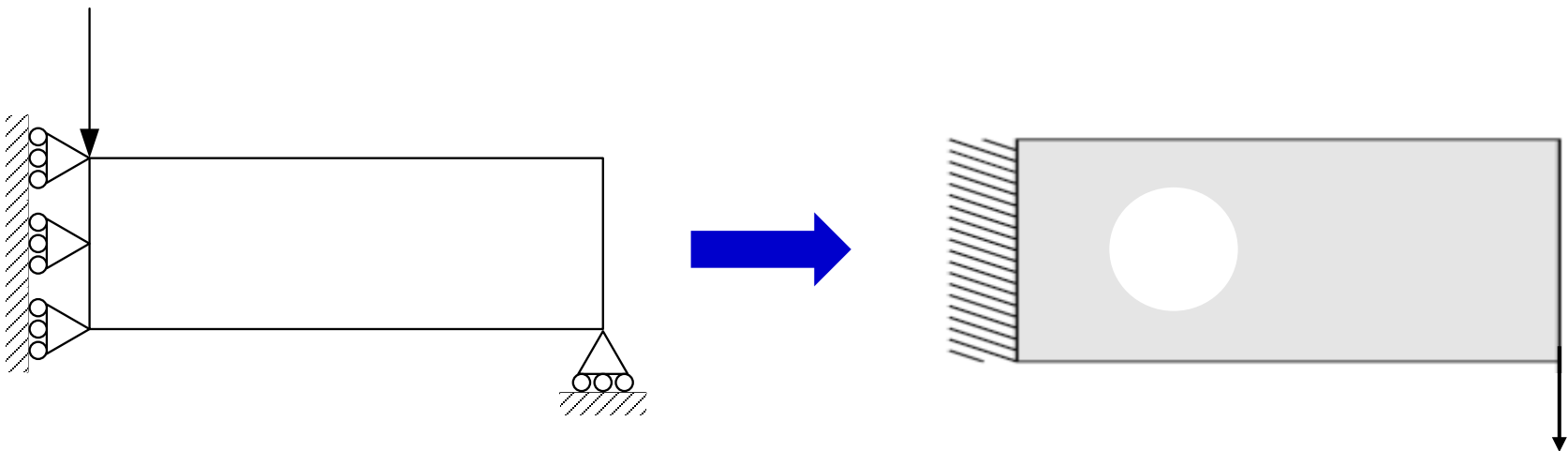


■ Exercise



▪ Extensions: Passive element

- Center of void is $(nely/2, nelx/3)$ and radius with $nely/3$



▪ Extensions: Passive element

Passive element

A $n_{ely} \times n_{elx}$ matrix **passive** is defined with 0 at elements free to change, 1 at elements fixed to be void, And 2 at elements fixed to be solid.

```

22 # Define Passive elements
23 passive = np.zeros((nely,nelx), dtype=int)
24 for i in range(nelx):
25     for j in range(nely):
26         if np.sqrt((j-nely/2+1)**2+(i-nelx/3+1)**2) < nely/3:
27             passive[j,i] = 1

```

These lines must be inserted before the start of the optimization loop.

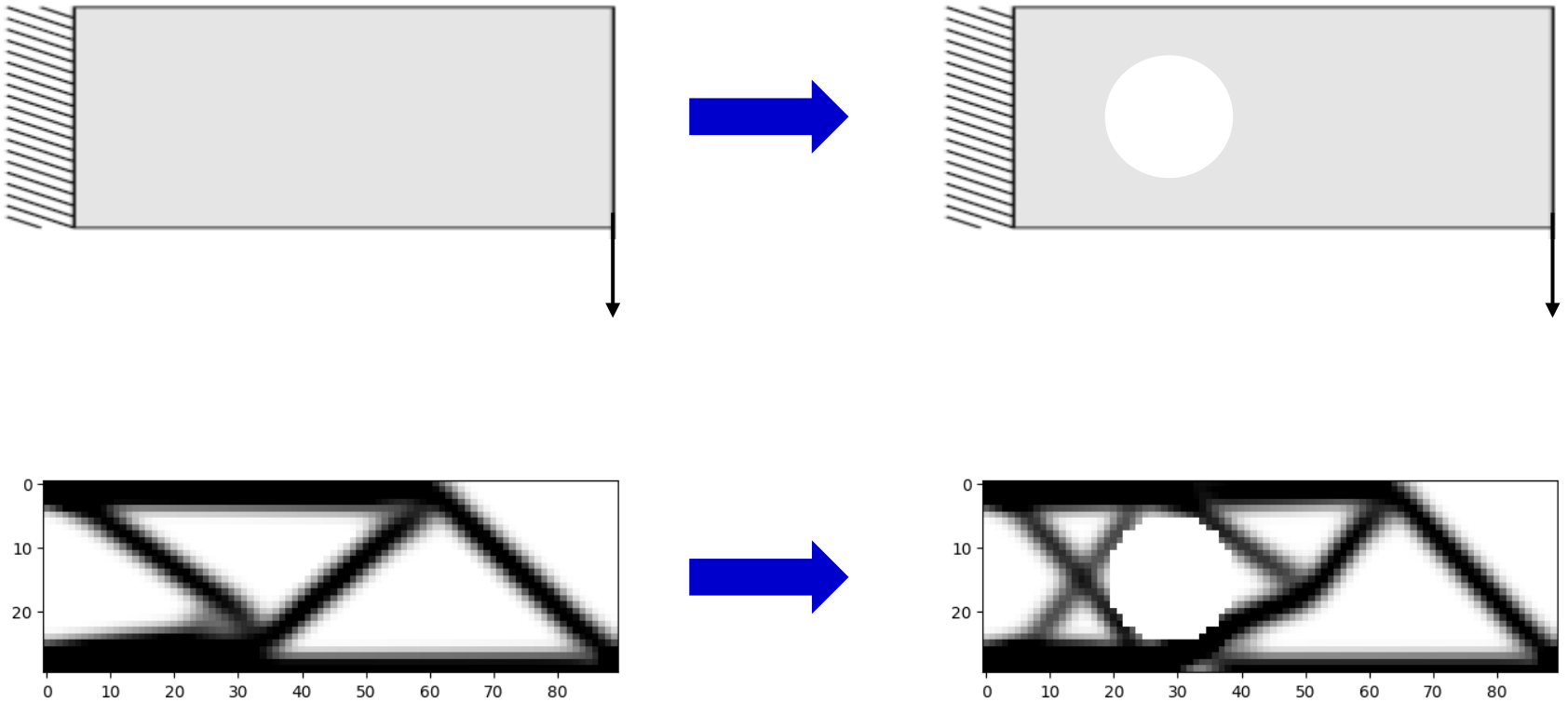
Then we can simply add two line in optimality criteria subroutines.

```

149 xnew[nely*np.where(passive==1)[1]+np.where(passive==1)[0]+1] = 0
150 xnew[nely*np.where(passive==2)[1]+np.where(passive==2)[0]+1] = 1

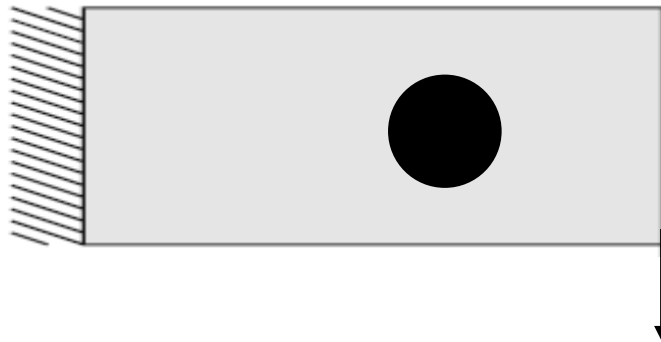
```

- Extensions: Passive element



■ Exercise

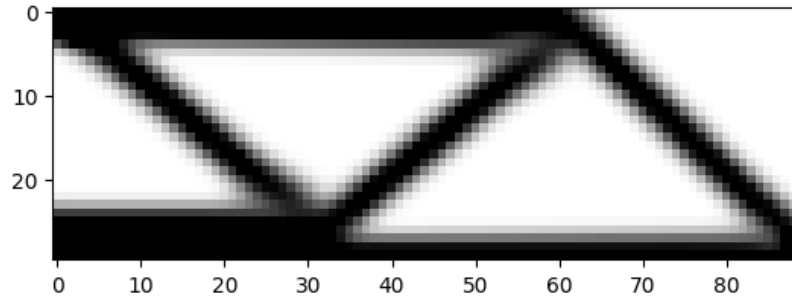
- Center of solid is $(ne1y/2, ne1y \times 2)$ and radius with $ne1x/10$



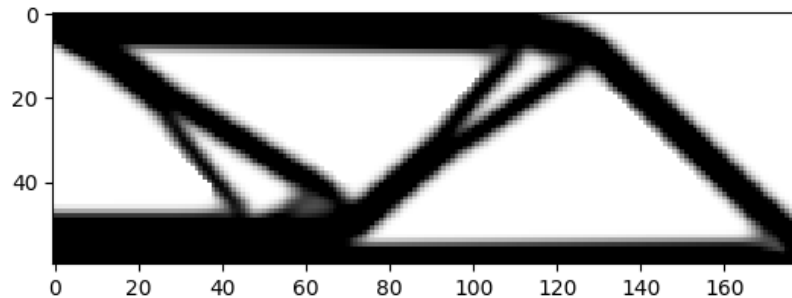
■ Parameter test

- `main(90,30,0.4,3.0,3.0,1)`
- `main(180,60,0.4,3.0,3.0,1)` ← To check mesh-independency
- `main(90,30,0.4,3.0,0.1,1)` ← To check checkerboard pattern
- `main(90,30,0.7,3.0,3.0,1)` ← To check volume fraction ratio
- `main(90,30,0.5,1.0,3.0,1)` ← To check penalty term

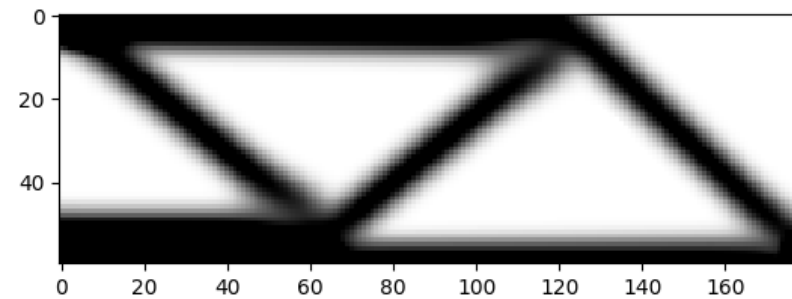
■ Parameter test: Mesh dependency



➤ `main(90,30,0.4,3.0,1.5,1)`

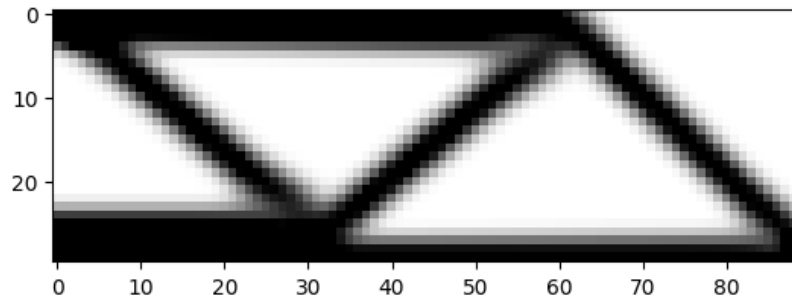


➤ `main(180,60,0.4,3.0,1.5,1)`

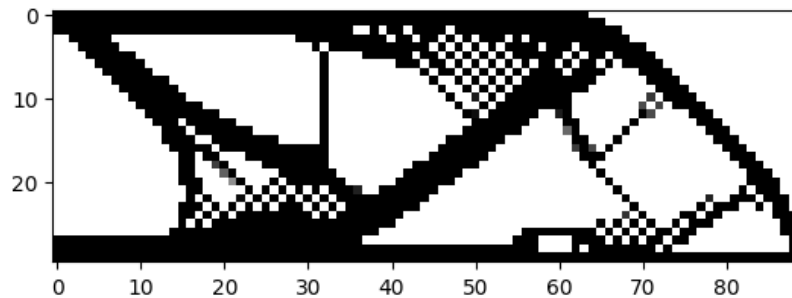


➤ `main(180,60,0.4,6.0,1.5,1)`

Parameter test: Checkerboard pattern

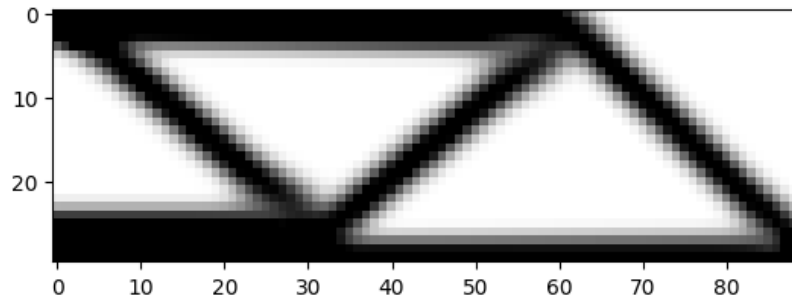


➤ `main(90,30,0.4,3.0,1.5,1)`

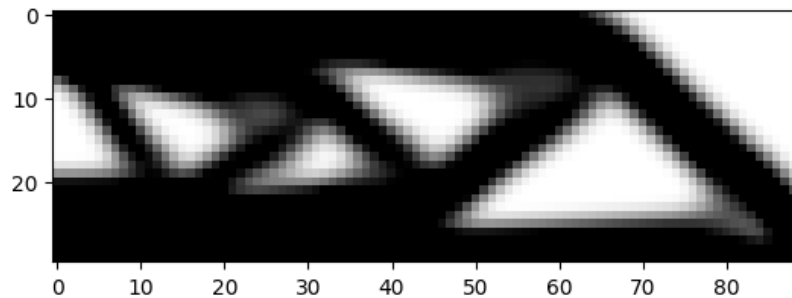


➤ `main(90,30,0.4,3.0,0.1,1)`

- Parameter test: Volume fraction ratio

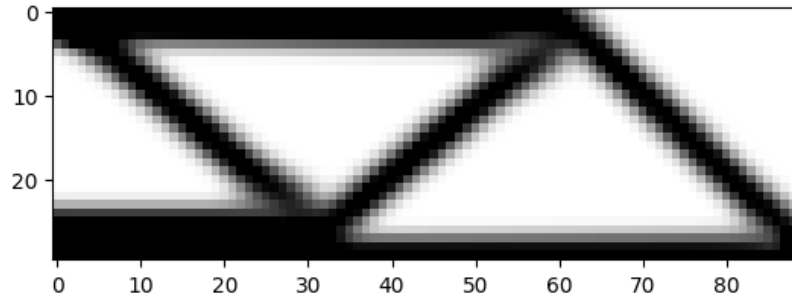


➤ `main(90,30,0.4,3.0,1.5,1)`

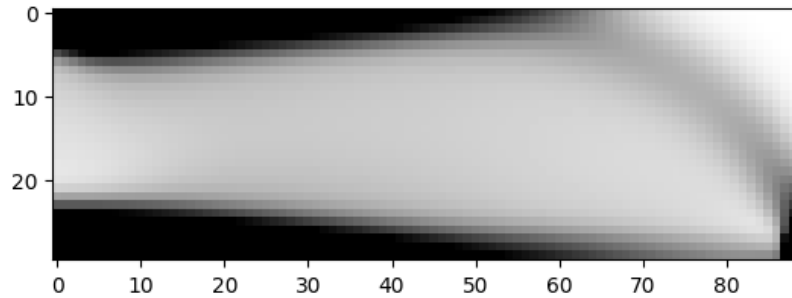


➤ `main(90,30,0.7,3.0,0.1,1)`

■ Parameter test: Penalty term



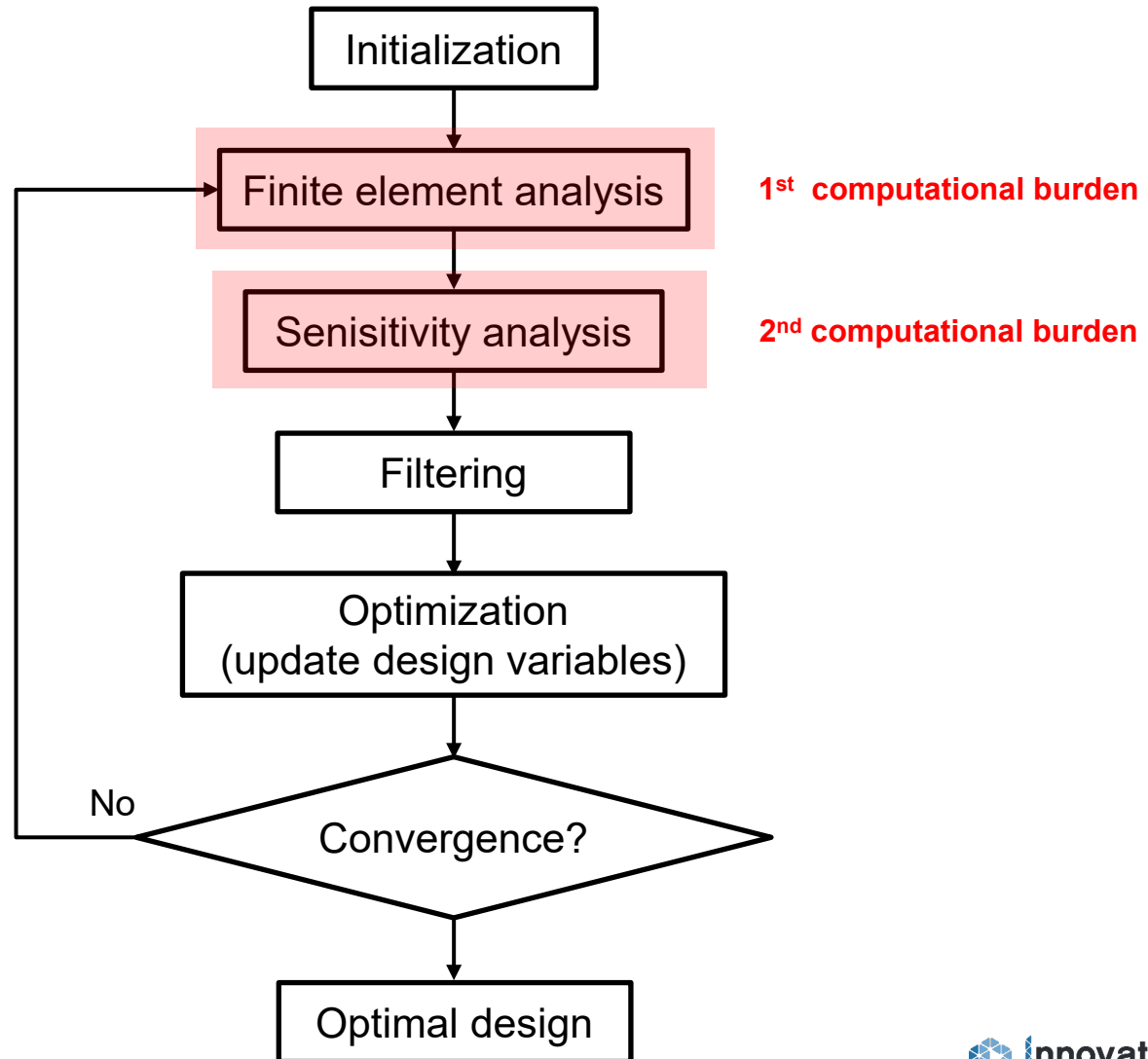
➤ `main(90,30,0.4,3.0,1.5,1)`



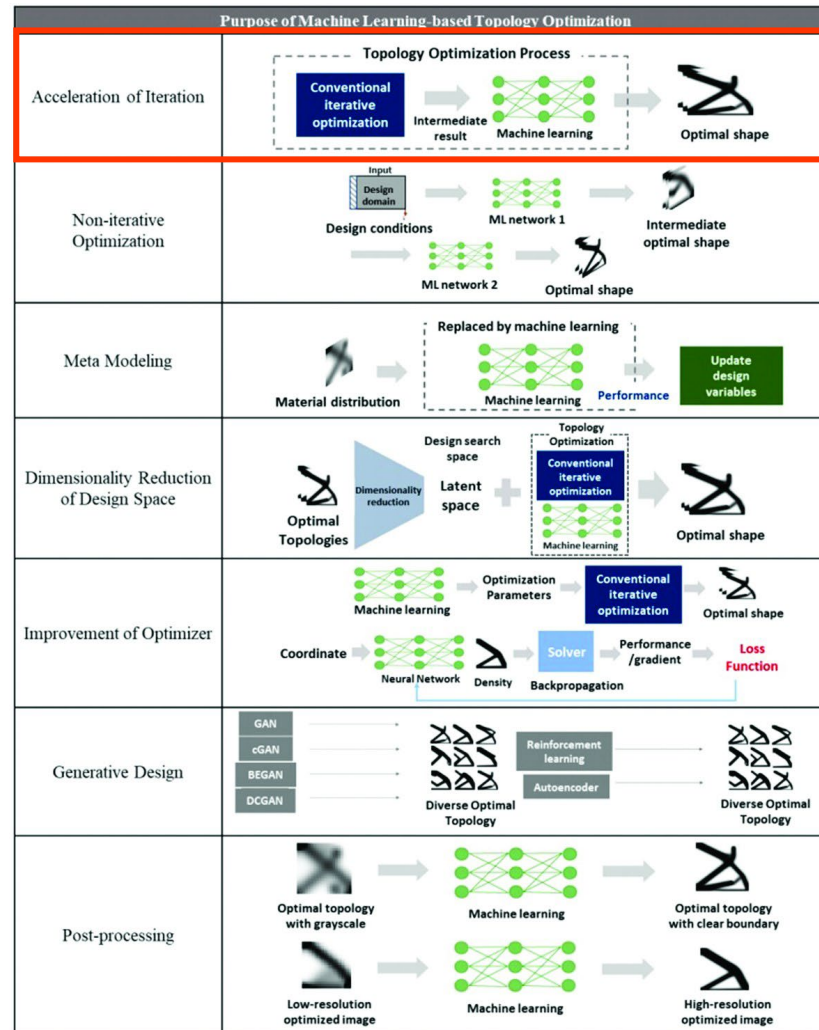
➤ `main(90,30,0.4,1.0,3.0,1)`

Acceleration of topology optimization using artificial intelligence

- Acceleration of topology optimization using artificial intelligence



Acceleration of topology optimization using artificial intelligence



*S.Shin, D. Shin, N. Kang, "Topology optimization via machine learning and deep learning: a review", *Journal of Computational Design and Engineering*, vol. 10, no. 4, pp. 1736-1766, 2023.

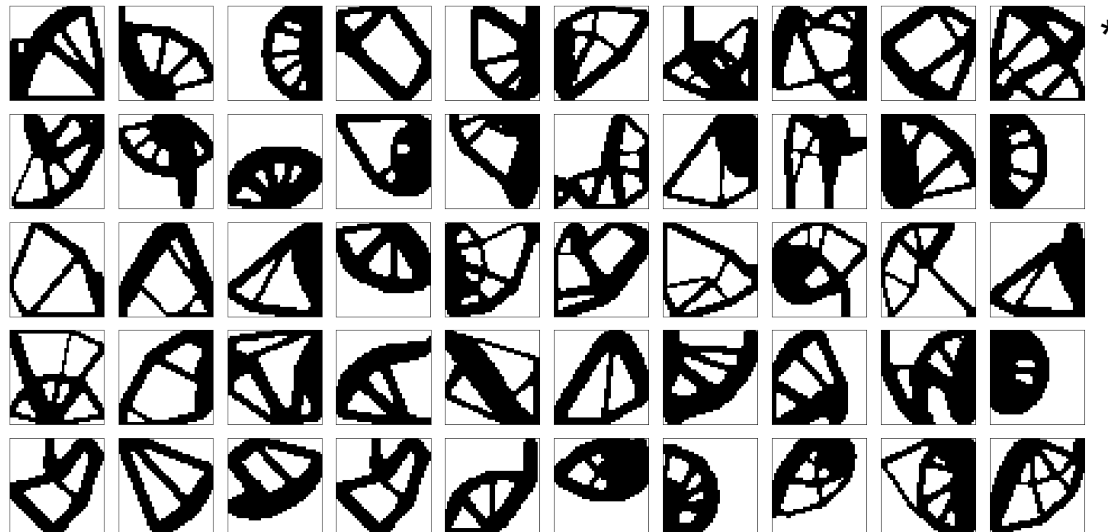
■ Dataset

- The number of nodes with fixed x and y translations and the number of loads are sampled from the Poisson distribution:

$$N_x \sim P(\lambda = 2)$$

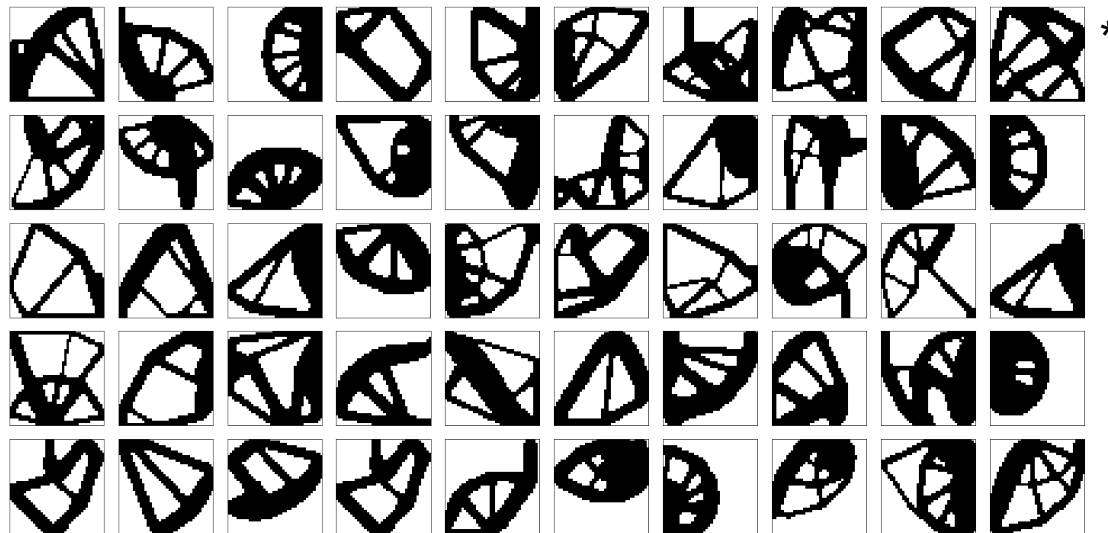
$$N_y, N_L \sim P(\lambda = 1).$$

- The nodes for each of the above-described constraints are sampled from the distribution defined on the grid. The probability of choosing the boundary node is **100 times higher** than that for an inner node.



■ Dataset

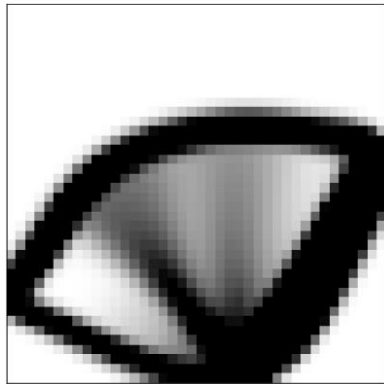
- The load values are chosen as -1 .
- The volume fraction is sampled from the Gaussian distribution with **mean of 0.5 and std of 0.1**
- The obtained dataset has 10,000 objects. 7,000 objects are allocated for training and 3,000 objects are allocated for testing.
- Each object is a tensor of shape $2 \times 40 \times 40$: **7, 100 iterations of the optimization process** for the problem defined on a regular 40×40 grid.



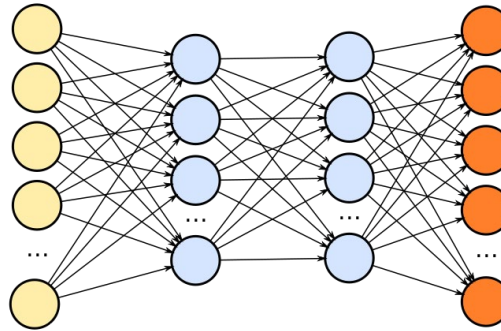
*I. Sosnovik, I. Oseledets, "Neural networks for topology optimization", *Russian Journal of Numerical Analysis and Mathematical Modelling*, vol.34, no. 4, pp. 215-223, 2019.

■ Deep learning architecture

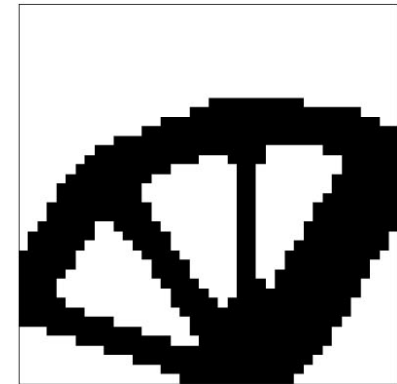
- PyTorch is used as a deep learning framework.
- Training is conducted with 20 fixed epochs, 16 batch size, MAE loss, Adam optimizer.



Intermediate result
(iter 7)

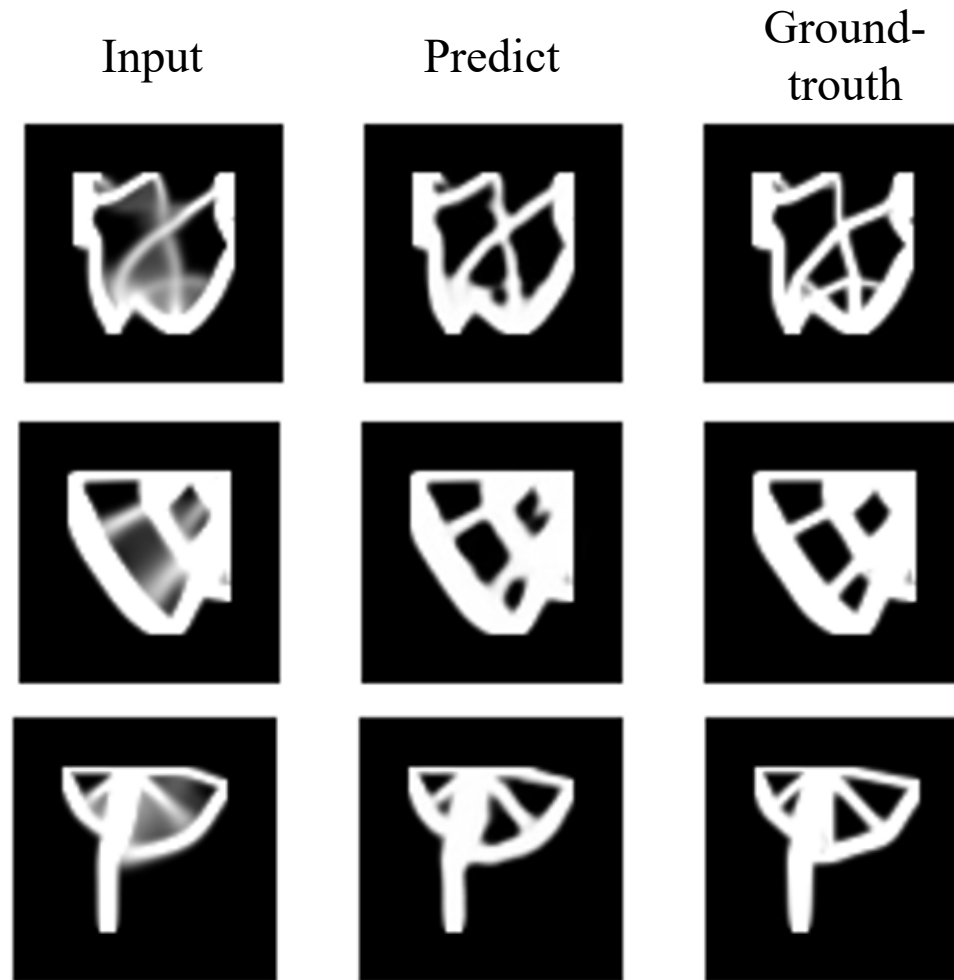


Deep neural network
(Unet)



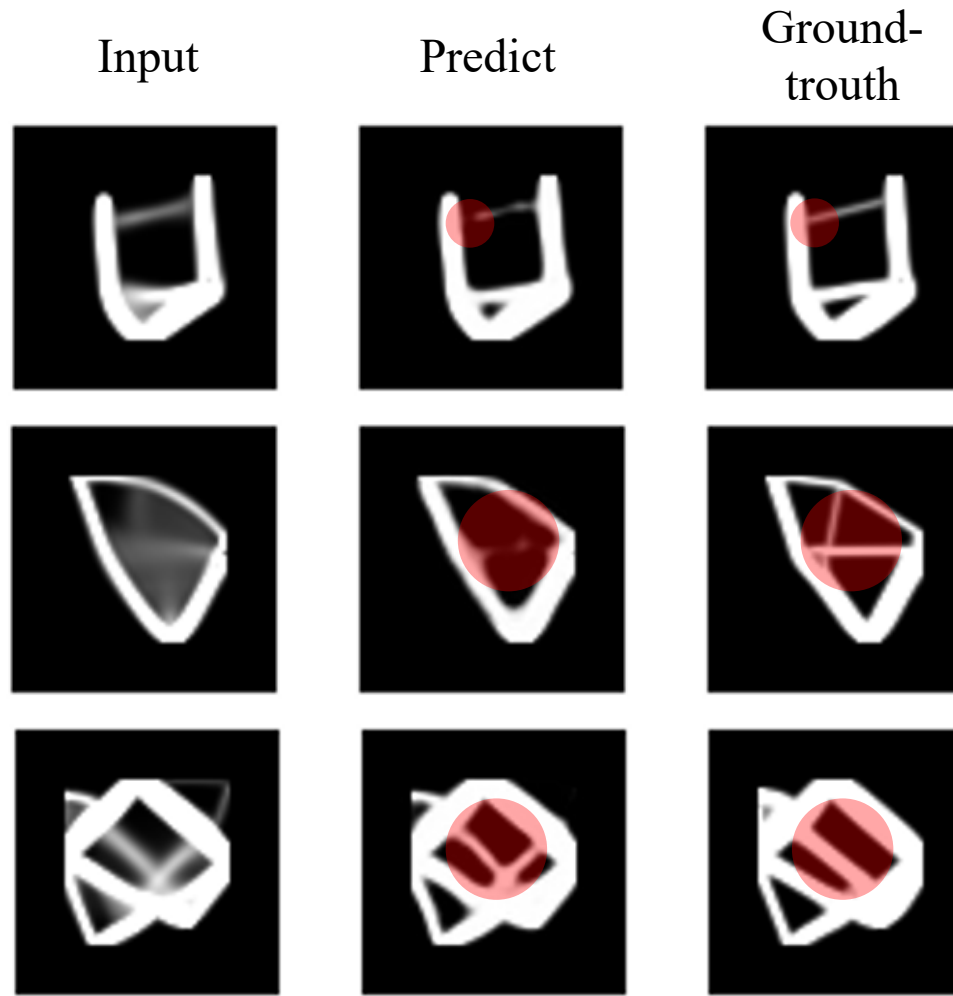
Ground truth
(iter 100)

■ Result



*S.Shin, D. Shin, N. Kang, "Topology optimization via machine learning and deep learning: a review", *Journal of Computational Design and Engineering*, vol. 10, no. 4, pp. 1736-1766, 2023.

■ Result



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Thank you!

- Contact: hyukjin.koh@kaist.ac.kr

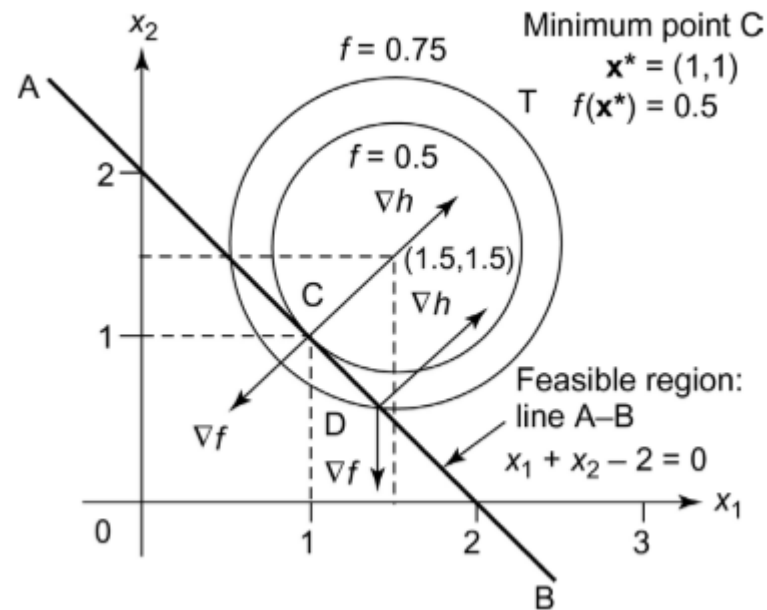
■ Lagrange multiplier

Minimize

$$f(x_1, x_2) = (x_1 - 1.5)^2 + (x_2 - 1.5)^2$$

subject to an equality constraint:

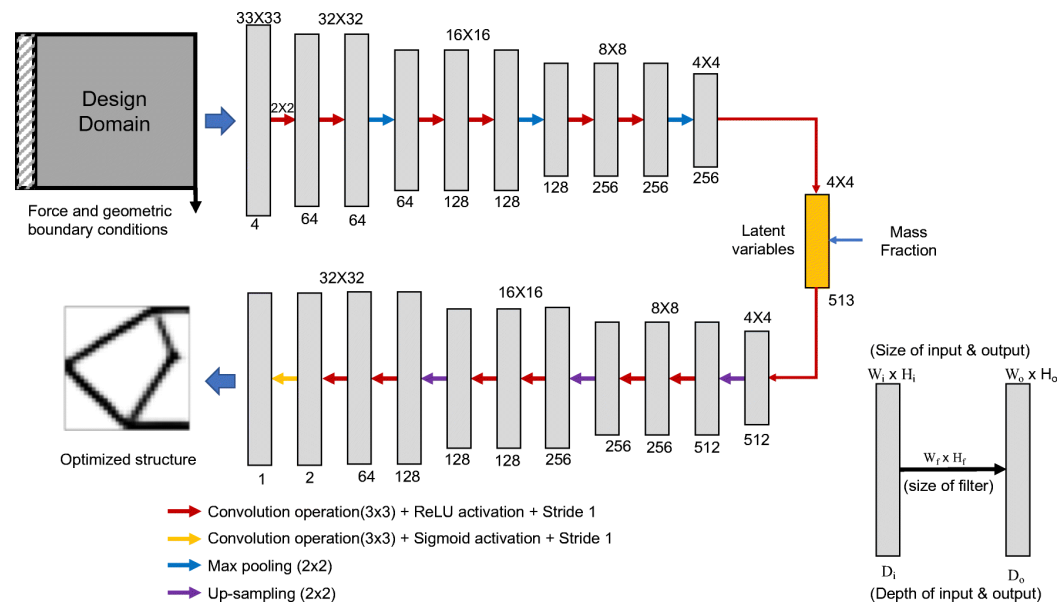
$$h(x_1, x_2) = x_1 + x_2 - 2 = 0$$



*Yu Y, Hur T, Jung J, Jang IG (2019) Deep learning for determining a near-optimal topological design without any iteration. Struct Multidisc Optim 59(3):787–799. <https://doi.org/10.1007/s00158-018-2101-5>

■ Direct design

- The direct design model approach is currently one of the most popular applications of AI in TO, and the aim is to directly achieve an optimised structure for a given problem definition, **completely removing the need for expensive iterative procedures**.
- Commonly this is achieved by implementing neural network architectures popular in image segmentation, like CNN or GAN.



*Yu Y, Hur T, Jung J, Jang IG (2019) Deep learning for determining a near-optimal topological design without any iteration. Struct Multidisc Optim 59(3):787–799. <https://doi.org/10.1007/s00158-018-2101-5>

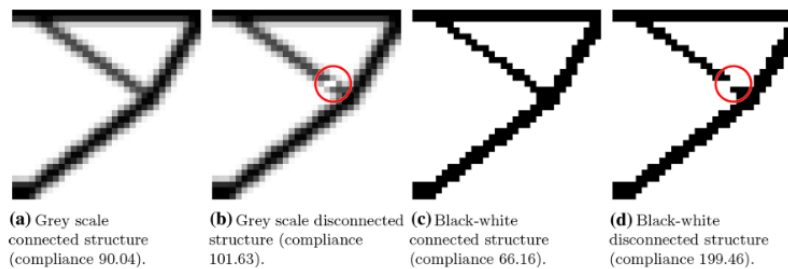
▪ Direct design: limitations

1. Mesh dependency

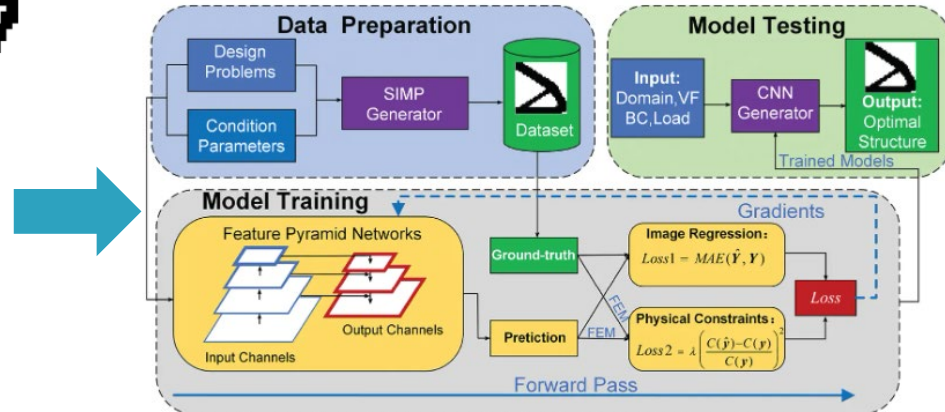
- The presented direct design models are typically **restricted to a small fixed mesh**.
- It is unclear whether the model is readily translated to problems with different mesh dimensions or resolutions, even given the inherent flexibility of the CNN.
- The network size increases with the number of elements in the mesh, resulting in more parameters to be determined during training and a larger memory consumption for storing the model. In turn, this also increases the cost of obtaining training samples, as finer meshes imply **more time needed to optimize a structure using conventional TO**.

2. Structural disconnection

- Image-based errors (e. g. MAE) not reflect the quality of a structure and thus the network learns based on an incorrect measure.



	Grey scale	Black-and-white
MAE	0.0014	0.0039
Gap	0.1288	2.0148

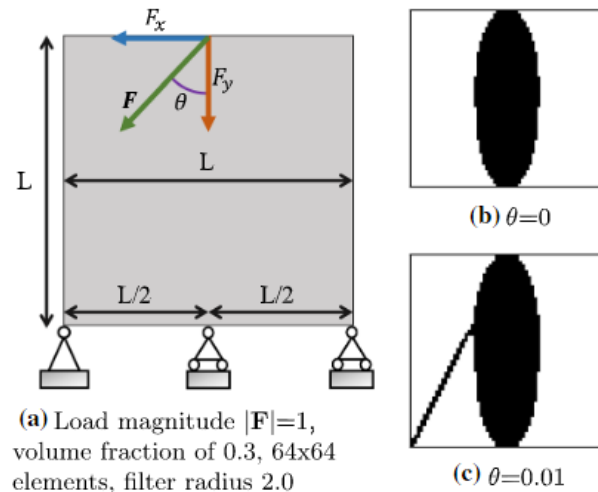


*Luo J, Li Y, Zhou W, Gong Z, Zhang Z, Yao W (2021) An Improved Data-Driven Topology Optimization Method Using Feature Pyramid Networks with Physical Constraints. Comput Model Eng Sci 128(3):823–848. <https://doi.org/10.32604/cmescs.2021.016737>, <https://www.techscience.com/CMES/v128n3/44011>

▪ Direct design: limitations

3. Vulnerability to perturbation (major)

- Relatively small changes to the boundary conditions can lead to a very different solution being optimal.
- Any learning-based approach would face the challenge that a small perturbation of the boundary conditions could lead to a big change in the optimal structure.
- Unless the types of problems are strictly limited, it is clear that an **unbounded number of examples could be necessary** to learn all the discontinuities in the mapping from latent space to mechanical structure.



Design	(b)	(c)] Compliance
$\theta=0$	7.255	7.372	
$\theta = 0.01$	1,649,351,760	7.5591	