

Magnet design for MedAustron

February 20, 2022

1 Analytical

1.1 Magnet type decision

The arguments for and against a H-type magnet are:

- + Mechanical rigid
- + Symmetrical
- Hard to get the beam pipe in and out

1.2 Aperture Height

The aperture height is given as the sum of the good field region $h_{\text{GFR}} = 2 \cdot \text{GFR}_y$, the thickness of the vacuum pipe d_{vacuum} and a tolerance for installation and thermal expansion $d_{\text{tolerance}}$ as

$$\begin{aligned} h &= h_{\text{GFR}} + 2 \cdot d_{\text{vacuum}} + d_{\text{tolerance}} \\ &= 2 \cdot 23 \text{ mm} + 2 \cdot 2 \text{ mm} + 2 \text{ mm} \\ &= 52 \text{ mm}. \end{aligned} \tag{1}$$

1.3 Flux Density

The total bending angle generated by all $m = 3$ magnets is

$$\theta_{\text{tot}} = 3 \cdot \theta_{\text{mag}} = 3 \cdot 36^\circ = 108^\circ. \tag{2}$$

The magnetic length of the (dipole) magnet can be approximated by

$$\begin{aligned} l_{\text{mag}} &= l_{\text{iron, max}} + 2hk \\ &= 0.340 \text{ m} + 2 \cdot 0.55 \cdot 52 \text{ mm} \\ &= 397.2 \text{ mm}. \end{aligned} \tag{3}$$

With the definition of the radian ($\theta = s/\rho$) and $s = l_{\text{mag}}$, the bending radius ρ is

$$\theta_{\text{mag}} = \frac{l_{\text{mag}}}{\rho} \tag{4}$$

$$\begin{aligned} \Rightarrow \rho &= \frac{l_{\text{mag}}}{\theta_{\text{mag}}} \\ &= \frac{397.2 \text{ mm}}{0.6283 \text{ rad}} \\ &= 0.642 \text{ m}. \end{aligned} \tag{5}$$

With the given minimum and maximum $(B\rho)$ for a proton and a C^{6+} beam, the minimum and maximum needed flux densities are

$$B_{\text{min}} = \frac{(B\rho)_{\text{min}}}{\rho} = \frac{0.383 \text{ T m}}{0.642 \text{ m}} = 0.596 \text{ T} \tag{6}$$

$$B_{\text{max}} = \frac{(B\rho)_{\text{max}}}{\rho} = \frac{0.766 \text{ T m}}{0.642 \text{ m}} = 1.19 \text{ T} \tag{7}$$

1.4 Pole width and yoke thickness

With the given field quality inside the GFR $\Delta B/B_0 = 1 \cdot 10^{-3}$, the optimized normalized pole overhang is¹

$$\begin{aligned} x_{\text{optimized}} &= 2 \frac{a_{\text{optimized}}}{h} \\ &= -0.14 \ln \left[\frac{\Delta B}{B_0} \right] - 0.25 \\ &= 0.7171. \end{aligned} \quad (8)$$

This equals a pole overhang of

$$\begin{aligned} a_{\text{optimized}} &= \frac{h \cdot x_{\text{optimized}}}{2} \\ &= \frac{52 \text{ mm} \cdot 0.7171}{2} \\ &= 18.645 \text{ mm}. \end{aligned} \quad (9)$$

Taking the sagitta s into account and using

$$\text{GFRx}' = \text{GFRx} + \underbrace{\rho(1 - \cos \theta/2)}_s = 20 \text{ mm} + 31.42 \text{ mm} = 51.42 \text{ mm}, \quad (10)$$

the pole width w computes to

$$\begin{aligned} w &= 2 \cdot (\text{GFRx}' + a_{\text{optimized}}) \\ &= 2 \cdot (51.42 \text{ mm} + 18.645 \text{ mm}) \\ &= 140.13 \text{ mm}. \end{aligned} \quad (11)$$

1.5 Excitation Current

The ampereturns are given as

$$(NI)_{\text{dipole, min}} = \frac{B_{\text{min}} h}{2 \mu_0} = \frac{0.596 \text{ T} \cdot 52 \text{ mm}}{2 \cdot \mu_0} = 12.331 \text{ kA} \quad (12)$$

$$(NI)_{\text{dipole, max}} = \frac{B_{\text{max}} h}{2 \mu_0} = \frac{1.19 \text{ T} \cdot 52 \text{ mm}}{2 \cdot \mu_0} = 24.621 \text{ kA} \quad (13)$$

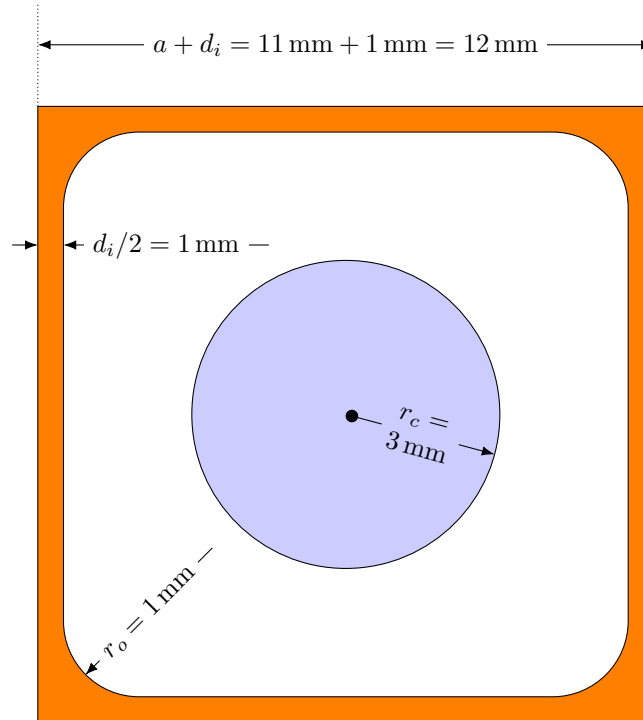


Figure 1: Cross section of the copper winding showing insulation(■) and water channel(■)

¹Compared to the unoptimized $x_{\text{unoptimized}} = 1.5898$

With the given dimensions of the copper winding (see Figure 1) and using

$$A_{\text{rounded edges}} = 4 \cdot A_{\text{rounded edge}} = 4 \frac{r_o^2(4 - \pi)}{4} = r_o^2(4 - \pi) \quad (14)$$

$$A_{\text{water channel}} = \pi r_c^2 \quad (15)$$

the copper cross section area is

$$\begin{aligned} A_{\text{copper}} &= a^2 - \underbrace{r_o^2(4 - \pi)}_{A_{\text{rounded edges}}} - \underbrace{\pi r_c^2}_{A_{\text{water channel}}} \\ &= 121 \text{ mm}^2 - 0.8584 \text{ mm}^2 - 28.274 \text{ mm}^2 \\ &= 91.867 \text{ mm}^2 \end{aligned} \quad (16)$$

With the given maximum current density of $j_{\text{max}} = 6 \text{ A mm}^{-2}$, the maximum possible current in the winding is

$$\begin{aligned} I_{\text{max}} &= j_{\text{max}} \cdot A_{\text{copper}} \\ &= 6 \text{ A mm}^{-2} \cdot 91.867 \text{ mm}^2 \\ &= 551.202 \text{ A}. \end{aligned} \quad (17)$$

With the calculated ampereturns, the necessary minimum number of turns are then

$$N_{B_{\text{min}}} = \left\lceil \frac{(NI)_{\text{min}}}{I_{\text{max}}} \right\rceil = 23 \quad (18)$$

$$N_{B_{\text{max}}} = \left\lceil \frac{(NI)_{\text{max}}}{I_{\text{max}}} \right\rceil = 45 \quad (19)$$

As it is easier to lower the current then to change the number of turns, the minimum number of turns should be $N_{B_{\text{max}}}$.

To full fill the empirical good practice $N_{\text{horizontal}}/N_{\text{vertical}} = 2$, the number of turn is set to

$$N = N_{\text{horizontal}} \times N_{\text{vertical}} = 10 \times 5 = 50. \quad (20)$$

With the new $N = 50$, the winding current is

$$I = \frac{(NI)_{\text{max}}}{50} = 492.42 \text{ A} \quad (21)$$

1.6 Coil Parameters

The coil width and height are given by (see Figure 2)

$$w_{\text{coil}} = N_{\text{horizontal}} \cdot (a + d_{\text{insulation}}) + 2 \cdot (d_{\text{epoxy}} + d_{\text{air}}) = 128 \text{ mm} \quad (22)$$

and

$$h_{\text{coil}} = N_{\text{vertical}} \cdot (a + d_{\text{insulation}}) + 2 \cdot (d_{\text{epoxy}} + d_{\text{air}}) = 68 \text{ mm} \quad (23)$$

using $N_{\text{horizontal}} = 10$, $N_{\text{vertical}} = 5$, $a = 11 \text{ mm}$, $d_{\text{insulation}} = 1 \text{ mm}$, $d_{\text{epoxy}} = 2 \text{ mm}$ and $d_{\text{air}} = 2 \text{ mm}$.

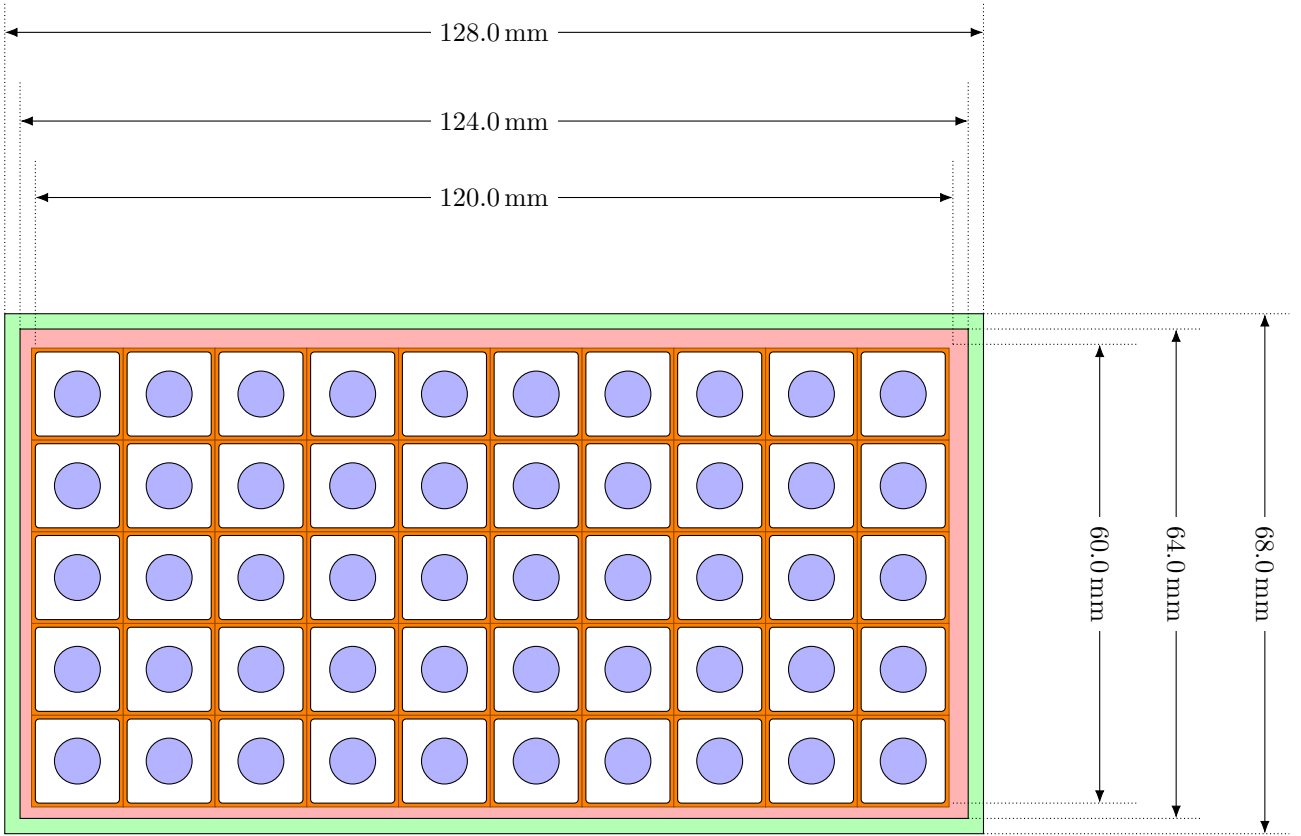


Figure 2: Cross section of one coil showing copper(□), insulation(■), the water channels(●), the coil epoxy coating(■) and the air gap around the coil(■)

With the pole perimeter

$$p = 2 \cdot l_{\text{iron}} + 2 \cdot w = 960 \text{ mm}, \quad (24)$$

the average turn length is ($l_{\text{iron}} = 340 \text{ mm}$, $w = 140.13 \text{ mm}$ and $w_{\text{coil}} = 128 \text{ mm}$)

$$l_{\text{avg}} = p + 4 \cdot w_{\text{coil}} = 1472 \text{ mm}. \quad (25)$$

The resistance of one coil winding is (with $1/\sigma_{\text{Cu}} = 1.72 \mu\Omega \text{ cm}$)

$$\begin{aligned} R_c &= \frac{N \cdot l_{\text{avg}}}{A \cdot \sigma_{\text{Cu}}} \\ &= \frac{50 \cdot 1472 \text{ mm}}{91.867 \text{ mm}^2 \cdot 1/1.72 \mu\Omega \text{ cm}} \\ &= 13.442 \text{ m}\Omega \end{aligned} \quad (26)$$

With the number of coils per magnet m , the DC steady-state voltage per magnet is

$$\begin{aligned} V_m &= I \cdot m R_c \\ &= 492.42 \text{ A} \cdot 2 \cdot 0.016 \Omega \\ &= 13.197 \text{ V}. \end{aligned} \quad (27)$$

The total voltage over all magnets is (number of magnets M)

$$\begin{aligned} V_{\text{total}} &= M \cdot V_m \\ &= 3 \cdot 13.197 \text{ V} \\ &= 39.591 \text{ V}. \end{aligned} \quad (28)$$

The dissipated power in one magnet is calculated to

$$\begin{aligned} P_m &= V_m \cdot I = I^2 \cdot m R_c \\ &= 13.197 \text{ V} \cdot 492.42 \text{ A} \\ &= 6.498 \text{ kW}. \end{aligned} \quad (29)$$

1.7 Cooling

With the given $\Delta T = 15 \text{ K}$ and the dissipated power $P_m = 6.498 \text{ kW}$, the total required water flow in one magnet is

$$Q = 14.3 \frac{P_m}{\Delta T} \cdot 1 \cdot 10^{-3} = 6.2 \text{ L min}^{-1} \quad (30)$$

Using the length of cooling circuit for one coil is ($K_c = 1$ and $K_w = 1$)

$$\begin{aligned} l &= \frac{K_c N l_{\text{avg}}}{K_w} \\ &= \frac{1 \cdot 50 \cdot 1472 \text{ mm}}{1} \\ &= 73.6 \text{ m.} \end{aligned} \quad (31)$$

The pressure drop over a cooling circuit is given by ($d = 2r_c = 6 \text{ mm}$)

$$\Delta p = 60 \cdot l \cdot \frac{Q^{1.75}}{d^{4.75}} \quad (32)$$

This can be interpreted a (non-linear) hydraulic equivalent to Ohm's law in the electrical domain:

$$\Delta p = \underbrace{\frac{60 \cdot l}{d^{4.75}}}_R \cdot Q^{1.75} = R \cdot Q^{1.75} \quad (33)$$

The hydraulic resistance of one coil is given (in AU) as

$$R = \frac{60 \cdot l}{d^{4.75}} = 0.888 \quad (34)$$

To cool one magnet, the cooling circuits can either be connected in series or in parallel (see Figure 3).

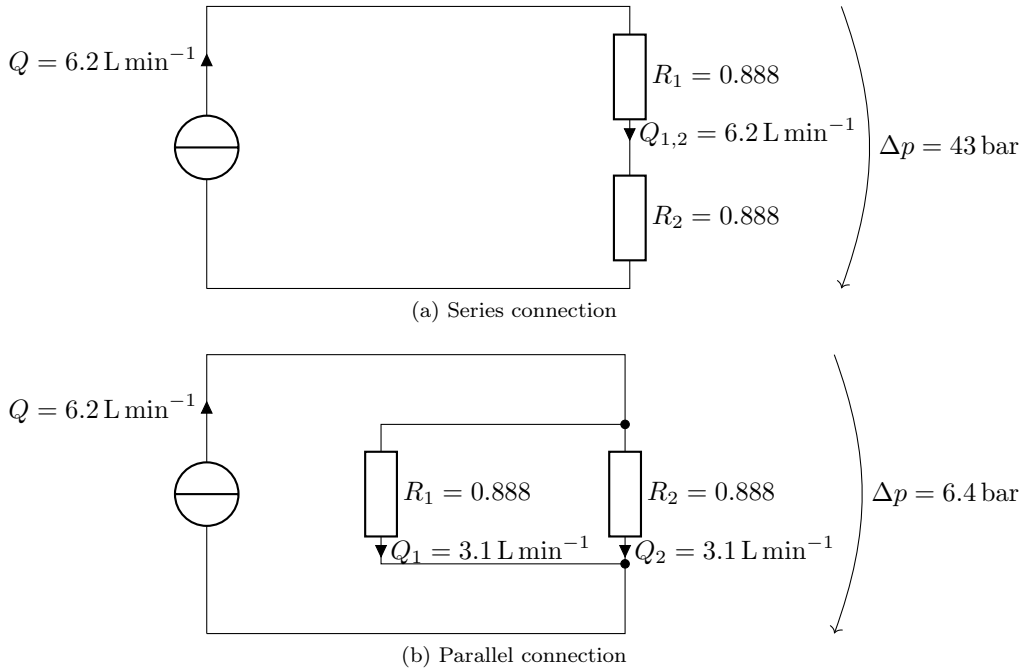


Figure 3: Possible cooling circuit configurations

For both cases, the pressure drops are

$$\Delta p_{\text{Series}} = 2 \cdot 0.888 \cdot 6.2^{1.75} = 43.26 \text{ bar} \quad (35)$$

$$\Delta p_{\text{Parallel}} = 0.888 \cdot 3.1^{1.75} = 6.43 \text{ bar} \quad (36)$$

As the series connection pressure drop exceeds the limit of the pump ($\Delta p_{\text{max}} = 7 \text{ bar}$), the parallel connection is used.

The average flow velocity in this case is

$$u_{\text{avg}} = 16.67 \cdot \frac{Q}{A} = 16.67 \cdot \frac{4 \cdot Q}{\pi d^2} = 16.67 \cdot \frac{4 \cdot 3.1 \text{ L min}^{-1}}{\pi \cdot (6 \text{ mm})^2} = 1.827 \text{ m s}^{-1}. \quad (37)$$

Reynolds number (with $\nu = 6.58 \cdot 10^{-7} \text{ m}^2 \text{ s}^{-1}$) is computed as

$$R_e = d \cdot \frac{u_{\text{avg}}}{\nu} \cdot 1 \cdot 10^{-3} = 16\,660. \quad (38)$$

From $R_e > 4000$ turbulent flow can be assumed for the flow in the water channels, which is needed for equal heat distribution.

2 Numerical

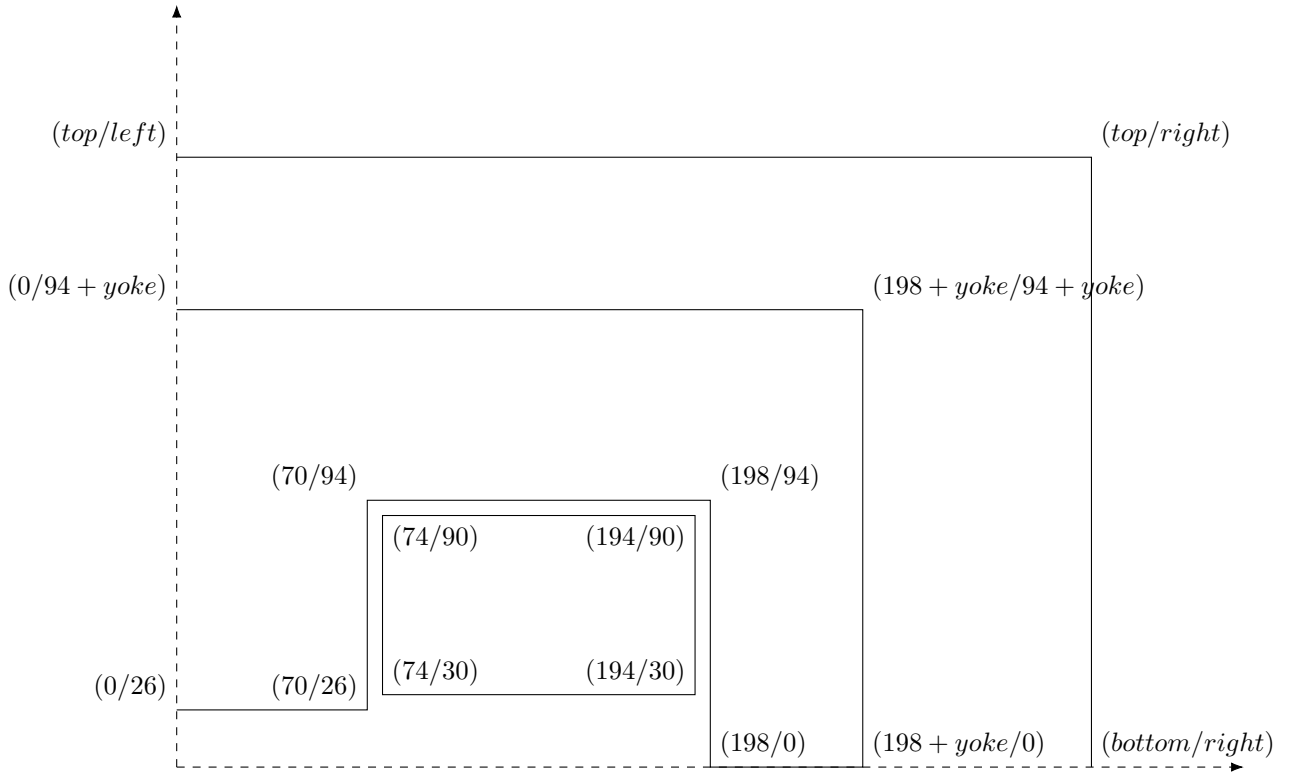


Figure 4: Input data for numerical simulation; all measurements in mm; drawing not to scale

Table 1: Relevant magnet parameters

Name	Value
Flux density B	999 T
Gap height h	999 mm
Pole width w	999 mm
Ampereturns NI	999 A