

## Part 1

Q1)

Theoretical

Step 1: Find the Transfer Function

As before, we can use the formula for the transfer function:

$$H(s) = C(sI - A)^{-1}B + D$$

Where  $s$  is the complex frequency variable, and  $I$  is the identity matrix.

## Code

```
%-----Q1-----  
% Define the open-loop transfer function G(s)  
num_G = 1;  
den_G = [1 1 0];    % s(s+1) = s^2 + s  
G_S = tf(num_G, den_G)  
  
% Define the feedback transfer function H(s)  
num_H = [1];  
den_H = [1];        % Unity Feedback  
H_S = tf(num_H, den_H)
```

## Output:

G\_S =

$$\frac{1}{s^2 + s}$$

Continuous-time transfer function.

H\_S =

$$1$$

Static gain.

Q2) the `step()` command to plot the output of  $G(s)$

Code: We made a function that plots step time response and checks stability

```
%-----Q2----- Step Response of G(s) (Open-Loop)
% Plot step response of G(s)
draw_step(G_S, 'Open-Loop System G(s)');

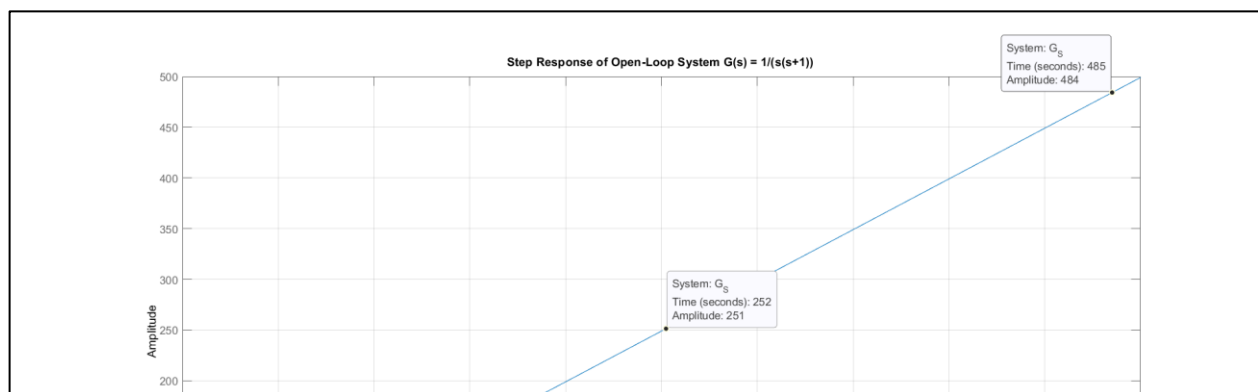
function draw_step(sys, sys_name)
% Create figure
figure;

% Plot step response
step(sys);
title(['Step Response of ', sys_name]);
grid on;

% Display poles
poles = pole(sys);
disp(['Poles of ', sys_name, ' :']);
disp(poles);

% Check stability
if all(real(poles) < 0)
    disp('System is stable (all poles in LHP)');
elseif any(real(poles) > 0)
    disp('System is unstable (at least one pole in RHP)');
else
    disp('System is marginally stable (poles on imaginary axis)');
end
end
```

Output:



Poles of Open-Loop System  $G(s)$ :

0  
-1

System is marginally stable (poles on imaginary axis)



Q3)

Code:

```
%-----Q3----- Closed-Loop Analysis
disp('Closed-Loop TF using feedback():');
T_feedback = feedback(G_S, H_S)

disp('Closed-Loop TF using manual formula (G/(1+GH)):');
T_manual = (1 / (1 + G_S * H_S)) * G_S; % Equivalent to T(s) = G/(1+GH)
T_manual = minreal(T_manual)           % Cancel common terms
```

Output:

```
Closed-Loop TF using feedback():

T_feedback =

      1
-----
s^2 + s + 1

Continuous-time transfer function.

Closed-Loop TF using manual formula (G/(1+GH)):

T_manual =

      1
-----
s^2 + s + 1

Continuous-time transfer function.
```

Q4)

Code:

```
%-----Q4----- Step Response of T(s) (Closed-Loop)
% Plot step response of T(s)
draw_step(T_feedback, 'Closed-Loop System T(s)');
```

Output:

	Before $G_c(s)$	with $G_c(s)$
$e_{s.s} _{\text{due to dist.}}$	$\frac{1}{2}$	zero
$\zeta$	0.4	0.69
$\omega_n$	14.8	58
$M_p$	25%	5%
$t_s$	0.67	0.1

Ouput:

System: Closed-Loop System T(s)

Poles: -0.5-0.86603i      -0.5+0.86603i

Stability: stable (all poles in LHP)

Over shoot MP: 16.2929% at t = 3.592 sec

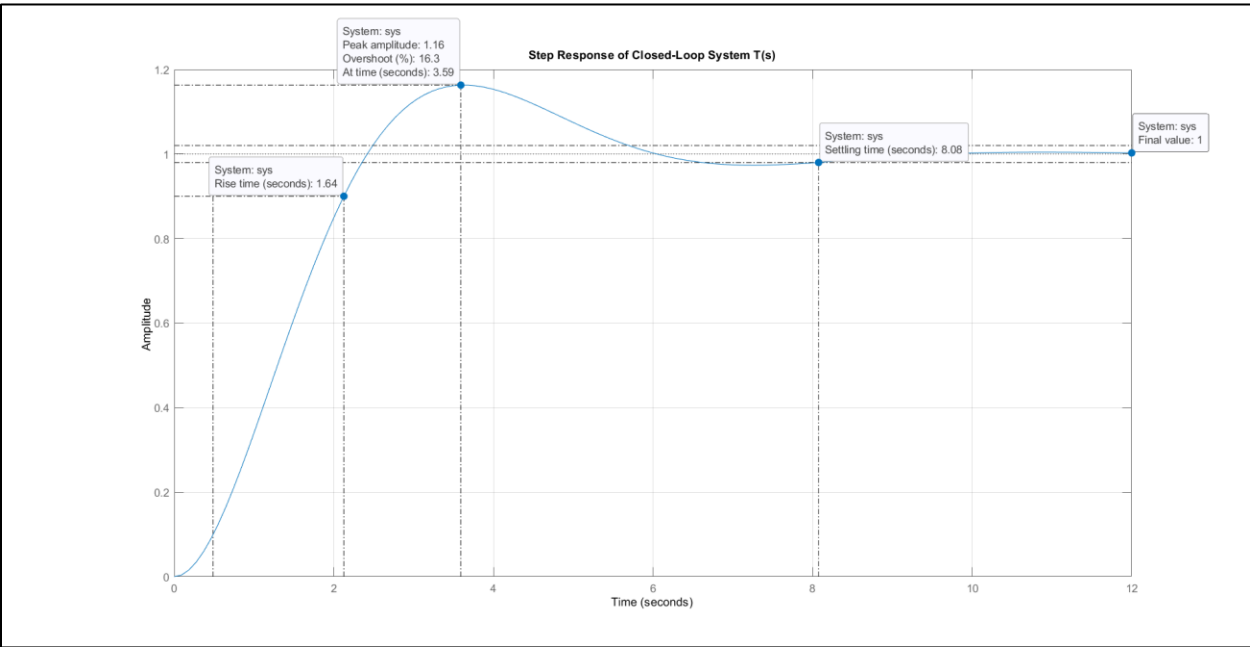
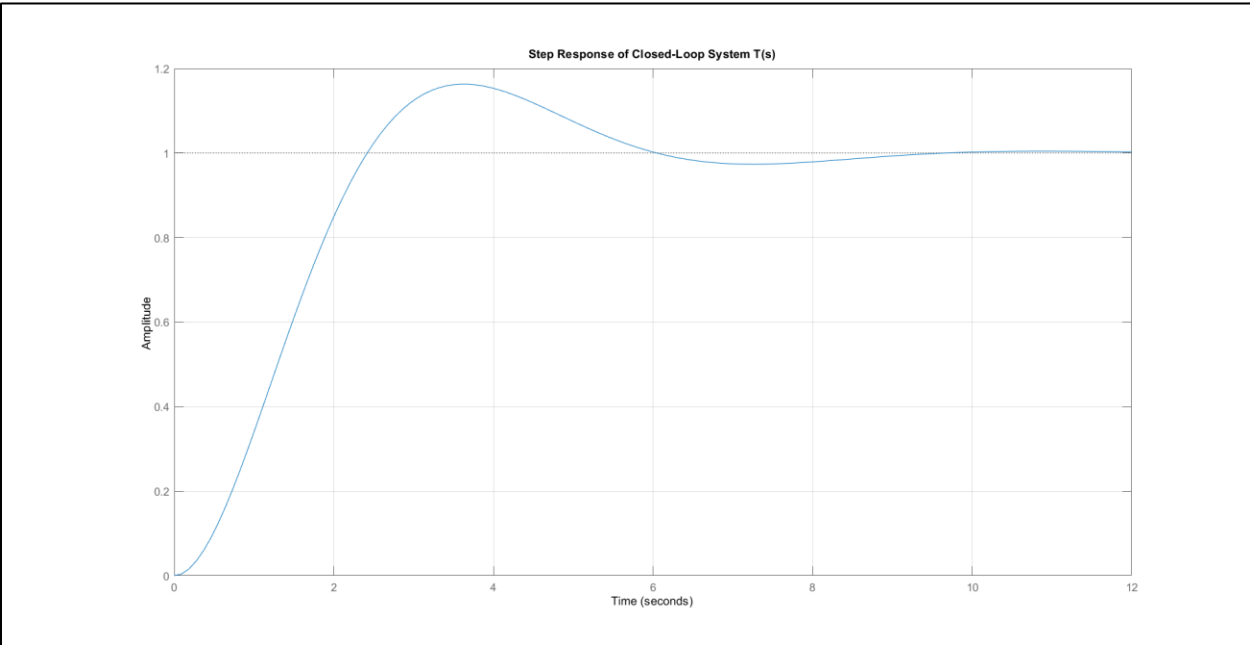
Damping ratio ( $\zeta$ ): 0.500

Natural frequency ( $\omega_n$ ): 1.000 rad/s

Settling time (2%): 8.1051 sec

Rise time (10-90%): 1.6579 sec

Steady-state value: 1.0014



Q5)

Code:

```
function [poles] = draw_poles(sys)

    % Create figure
    figure;

    % Plot pole-zero map
    pzmap(sys);
    title(['Pole-Zero Map of: ' inputname(1)]);
    grid on;

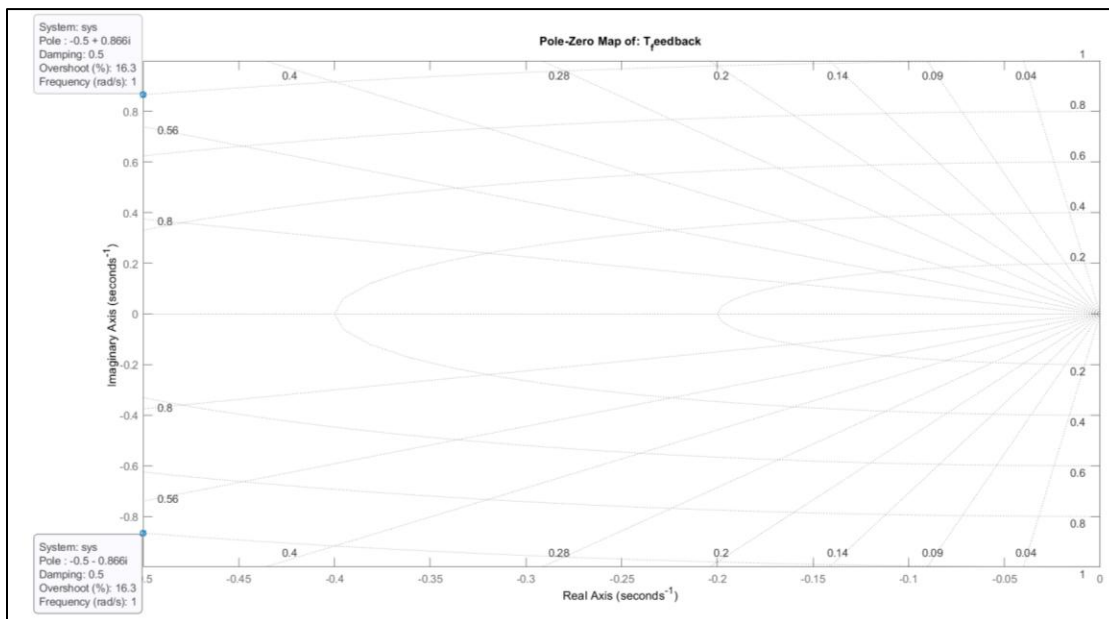
    % Get poles
    poles = pole(sys);

    % Display poles
    disp(['Poles of ' inputname(1) ':']);
    disp(poles);

    % Damping characteristics (for complex poles)
    if ~isreal(poles)
        [wn, zeta] = damp(sys);
        fprintf('Damping ratio (?): %.3f\n', zeta(1));
        fprintf('Natural frequency (?n): %.3f rad/s\n', wn(1));
    end

end
```

Output:





Poles of  $T_{\text{feedback}}$ :

$$-0.5000 + 0.8660i$$

$$-0.5000 - 0.8660i$$

Damping ratio ( $\zeta$ ): 0.500

Natural frequency ( $\omega_n$ ): 1.000 rad/s

Q6,Q7 is done

Q8

Code:

```
function [ess, t_out, y_out] = draw_ramp(sys, t_end, zoom_time)
{
    % Set defaults if not provided
    if nargin < 2
        t_end = 100;
    end
    if nargin < 3
        zoom_time = 700;
    end

    % Create time vector
    t = 0:0.1:t_end;

    %getting the ramp
    ramp = tf(1,[1 0]);

    % Get response data
    [y_sys, t_sys] = step(sys.*ramp, t);
    [y_ideal, t_ideal] = step(ramp, t);

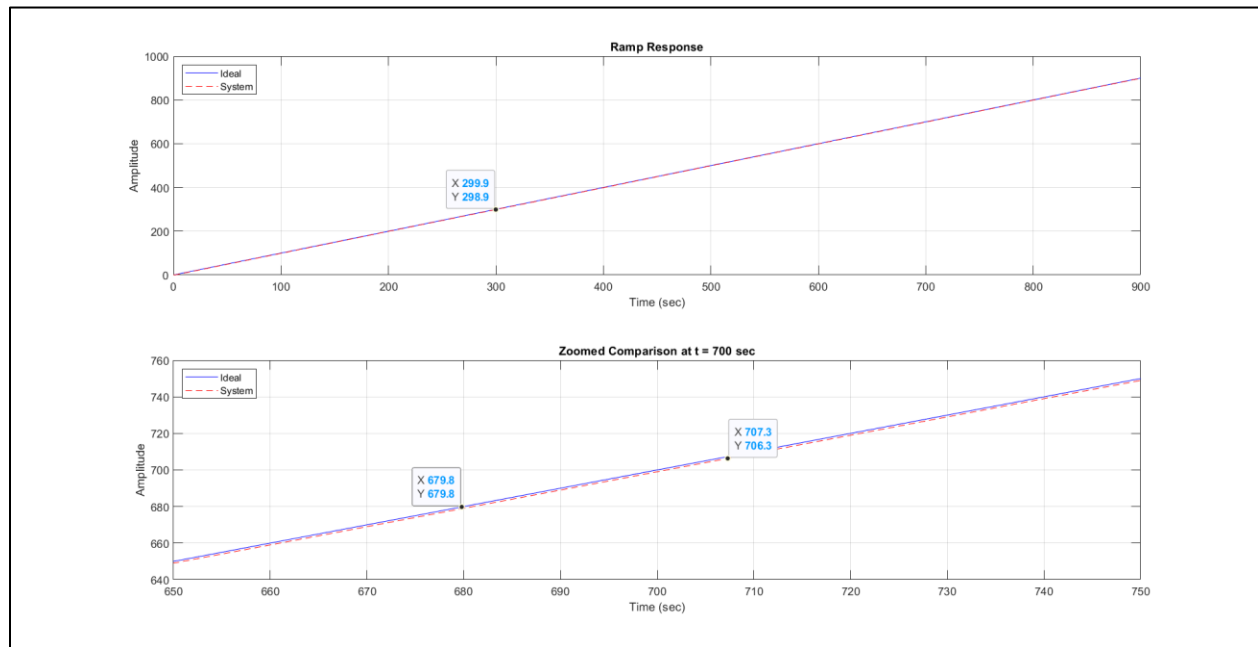
    % Create figure with three subplots
    figure;

    % Subplot 1: Ideal ramp input
    subplot(2,1,1);
    plot(t_ideal, y_ideal, 'b');
    hold on;
    plot(t_sys, y_sys, 'r--');
    title('Ramp Response');
    xlabel('Time (sec)');
    ylabel('Amplitude');
    legend('Ideal', 'System', 'Location', 'northwest');
    grid on;
    hold off;

    % Subplot 2: Zoomed comparison
    subplot(2,1,2);
    plot(t_ideal, y_ideal, 'b');
    hold on;
    plot(t_sys, y_sys, 'r--');
    xlim([zoom_time-50 zoom_time+50]);
    title(['Zoomed Comparison at t = ', num2str(zoom_time), ' sec']);
    xlabel('Time (sec)');
    ylabel('Amplitude');
    legend('Ideal', 'System', 'Location', 'northwest');
    grid on;
    hold off;

end
```

Output:



Steady-state error (ess): 1

Q9)

Code:

```
function [Gm, Pm, Wgc, Wpc] = draw_Bode_Plot(sys)
% BODE_PLOT Analyzes system stability margins and compares margin()
%   Bode_Plot(sys)
%
%   Input:
%   sys - Transfer function (tf object or state-space model)
%   Outputs:
%   Gm - Gain margin (dB)
%   Pm - Phase margin (degrees)
%   Wgc - Gain crossover frequency (rad/sec)
%   Wpc - Phase crossover frequency (rad/sec)

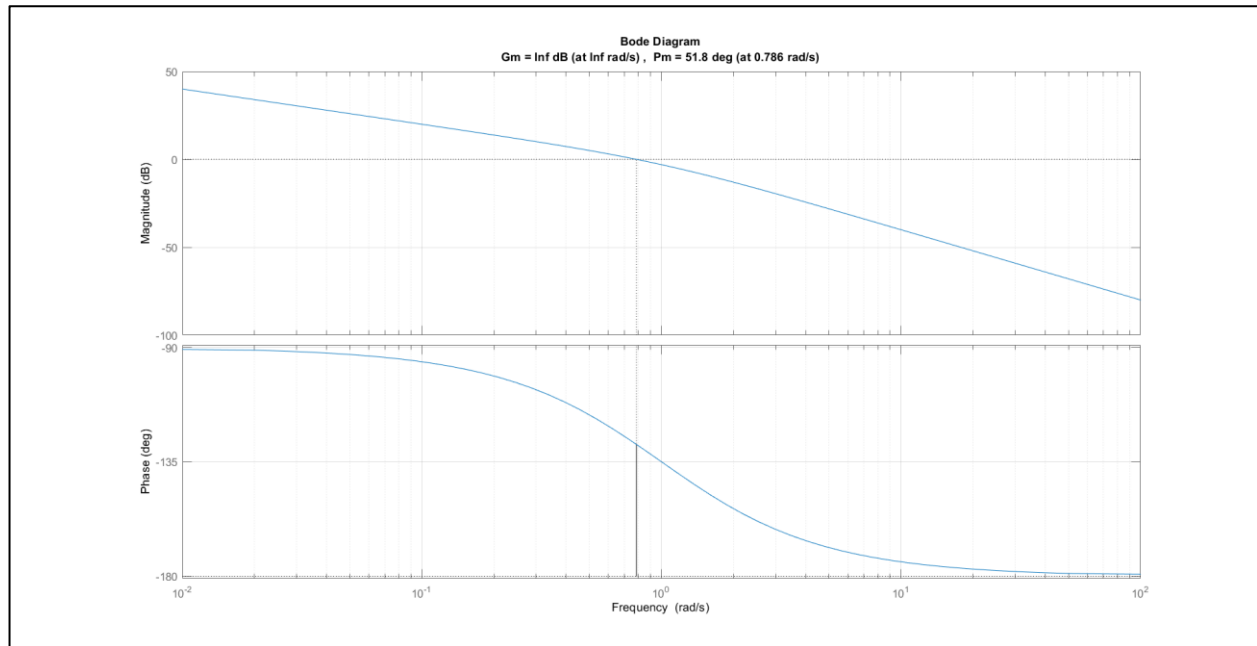
% Create margin plot
figure;
margin(sys);
grid on;

% Get stability margins
[Gm, Pm, Wgc, Wpc] = margin(sys);

% Display results
disp(['=== Stability Margins for ' inputname(1) ' ===']);
disp(['Gain Margin: ', num2str(Gm), ' dB at ', num2str(Wgc), ' rad/s']);
disp(['Phase Margin: ', num2str(Pm), '° at ', num2str(Wpc), ' rad/s']);

end
```

Output:



=== Stability Margins for ===

Gain Margin: Inf dB at Inf rad/s

Phase Margin: 51.8273° at 0.78615 rad/s

## Part2

Q2)

Code:

```
% Given system matrices
A = [0 1; -6 -5];
B = [0; 1];
C = [1 0];
D = [0];
n = 2; % System order
sys = ss(A,B,C,D); % State Space model
x0 = [0; 1]; % Initial condition

% Q2: Transfer function conversion
[num, den] = ss2tf(A,B,C,D);
syms s
TF_Manual = C*inv(s*eye(n)-A)*B + D;
TF_builtin = tf(num,den);
```

Output:

TF\_Manual =

$$1/(s^2 + 5s + 6)$$

TF\_builtin =

$$\frac{1}{s^2 + 5s + 6}$$

Continuous-time transfer function.

Q3:

Code

```
% Q3: State transition matrix calculation
% Compute  $\Phi(s) = [sI - A]^{-1}$ 
Phi_s = inv(s*eye(n) - A);

% Compute  $\Phi(t)$  by inverse Laplace transform
syms t
Phi_t = ilaplace(Phi_s);

% Verify  $\Phi(0) = I$ 
Phi_0 = subs(Phi_t, t, 0);

% Display results
disp('State transition matrix in s-domain ( $\Phi(s)$ ):');
pretty(Phi_s)

disp('State transition matrix in time domain ( $\Phi(t)$ ):');
pretty(Phi_t)

disp('Verification of  $\Phi(0) = I$ :');
disp(Phi_0);
```

## Output

State transition matrix in s-domain ( $\Phi(s)$ ):

$$\begin{bmatrix} \frac{1}{s+5} & \frac{1}{s+6} \\ \frac{6}{(s+5)^2} & \frac{s}{(s+5)(s+6)} \end{bmatrix}$$

State transition matrix in time domain ( $\Phi(t)$ ):

$$\begin{bmatrix} \exp(-2t) & \exp(-3t) - \exp(-2t) \\ \exp(-3t) & \exp(-2t) - \exp(-3t) \end{bmatrix}$$

Verification of  $\Phi(0) = I$ :

$$\begin{bmatrix} 1 & 0 \end{bmatrix}$$

$$\begin{bmatrix} 0 & 1 \end{bmatrix}$$



Q4)

Code:

```
% Q4: Verify that  $\Phi(t) = A\Phi(t)$ 
Phi_dot = diff(Phi_t, t); % Take time derivative of  $\Phi(t)$ 
A_Phi = A*Phi_t; % Multiply A with  $\Phi(t)$ 

disp('Time derivative of state transition matrix ( $\Phi(t)$ ):');
pretty(Phi_dot)

disp('A* $\Phi(t)$ :');
pretty(A_Phi)

disp('Verification successful:  $\Phi(t) = A\Phi(t)$ ');
```

Output:

```
Time derivative of state transition matrix ( $\Phi(t)$ ):
/ exp(-3 t) 6 - exp(-2 t) 6, exp(-3 t) 3 - exp(-2 t) 2 \
|
\ exp(-2 t) 12 - exp(-3 t) 18, exp(-2 t) 4 - exp(-3 t) 9 /

A* $\Phi(t)$ :
/ exp(-3 t) 6 - exp(-2 t) 6, exp(-3 t) 3 - exp(-2 t) 2 \
|
\ exp(-2 t) 12 - exp(-3 t) 18, exp(-2 t) 4 - exp(-3 t) 9 /

Verification successful:  $\Phi(t) = A\Phi(t)$ 
```

Q5)

Code:

```
% Q5 Check Controllability and Observability
% Check Controllability
Co = ctrb(A, B); % Controllability matrix
rank_Co = rank(Co);
disp('Controllability Matrix:');
disp(Co);
disp(['Rank of Controllability Matrix: ', num2str(rank_Co)]);

if rank_Co == n
    disp('System is Controllable (as expected)');
else
    disp('System is Not Controllable (unexpected for this system)');
end

% Check Observability
Ob = obsv(A, C); % Observability matrix
rank_Ob = rank(Ob);
disp('Observability Matrix:');
disp(Ob);
disp(['Rank of Observability Matrix: ', num2str(rank_Ob)]);

if rank_Ob == n
    disp('System is Observable (as expected)');
else
    disp('System is Not Observable (unexpected for this system)');
end
```

Output:

Controllability Matrix:

$$\begin{bmatrix} 0 & 1 \end{bmatrix}$$
$$\begin{bmatrix} 1 & -5 \end{bmatrix}$$

Rank of Controllability Matrix: 2

System is Controllable (as expected)

Observability Matrix:

$$\begin{bmatrix} 1 & 0 \end{bmatrix}$$
$$\begin{bmatrix} 0 & 1 \end{bmatrix}$$

Rank of Observability Matrix: 2

System is Observable (as expected)

Q6)

Code:

```
% Q6: Unforced (Homogeneous) Response
disp('=== Unforced Response Analysis ===');

% Compute state solution x(t) = ?(t)*x0
x_t = Phi_t * x0;

disp('Unforced state solution x(t):');
pretty(x_t)

% Compute output solution y(t) = C*x(t) + D*u(t)
% Since u(t)=0 for unforced response:
y_t = C*x_t + D*0;

disp('Unforced output response y(t):');
pretty(y_t)

% Plot the results
t_vals = linspace(0, 5, 500); % Time vector from 0 to 5 seconds

% Convert symbolic expressions to numeric functions
x1_func = matlabFunction(x_t(1));
x2_func = matlabFunction(x_t(2));
y_func = matlabFunction(y_t);

% Evaluate solutions
x1_vals = arrayfun(x1_func, t_vals);
x2_vals = arrayfun(x2_func, t_vals);
y_vals = arrayfun(y_func, t_vals);

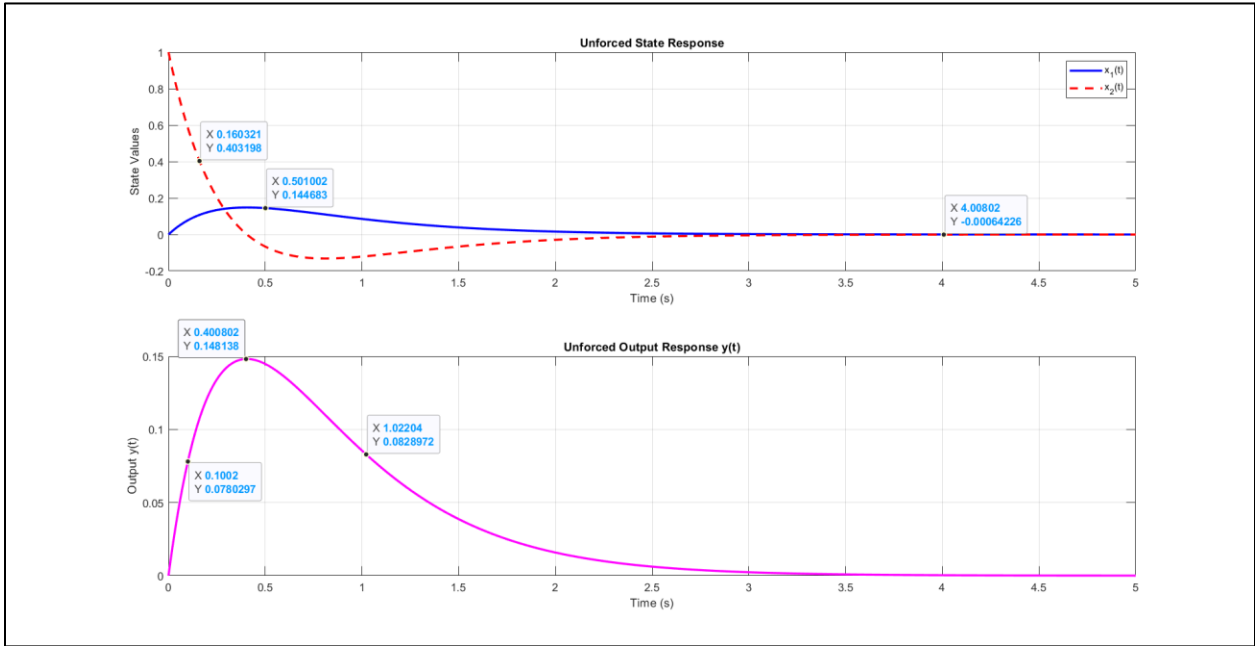
% Plot state responses
figure;
subplot(2,1,1);
plot(t_vals, x1_vals, 'b', 'LineWidth', 2);
hold on;
plot(t_vals, x2_vals, 'r--', 'LineWidth', 2);
title('Unforced State Response');
xlabel('Time (s)');
ylabel('State Values');
legend('x_1(t)', 'x_2(t)');
grid on;

% Plot output response
subplot(2,1,2);
plot(t_vals, y_vals, 'm', 'LineWidth', 2);
title('Unforced Output Response y(t)');
xlabel('Time (s)');
ylabel('Output y(t)');
grid on;

% Compare with MATLAB's built-in initial() function
[~,t_num,x_num] = initial(sys,x0,t_vals(end));
y_num = x_num*C'; % Equivalent to C*x since D=0

% Display symbolic solutions
disp(' ');
disp('Analytic Solutions:');
disp('x1(t) = '); pretty(x_t(1))
disp('x2(t) = '); pretty(x_t(2))
disp('y(t) = '); pretty(y_t)
```

Output:



=== Unforced Response Analysis ===

Unforced state solution  $x(t)$ :

$$\frac{\exp(-2t) - \exp(-3t)}{\exp(-3t) \cdot 3 - \exp(-2t) \cdot 2}$$

Unforced output response  $y(t)$ :

$$\exp(-2t) - \exp(-3t)$$

Analytic Solutions:

$$x_1(t) = \exp(-2t) - \exp(-3t)$$

$$x_2(t) = \exp(-3t) \cdot 3 - \exp(-2t) \cdot 2$$

$$y(t) = \exp(-2t) - \exp(-3t)$$

Q7)

Code:

```
% Q7: Forced Response Analysis (Unit Step Input)
disp('=== Forced Response Analysis ===');
% Using Frequency Domain Approach
U_s = 1/s; % Laplace transform of unit step
U_t = ilaplace(U_s);

% Compute forced component in frequency domain
X_forced_s = Phi_s * B * U_s;

% Convert to time domain
x_forced_t = ilaplace(X_forced_s);

% Total solution (homogeneous + forced)
x_total_t = x_t + x_forced_t;

% Output solution
y_total_t = C*x_total_t + D*U_t; % D*u(t) where u(t)=1 for t>0

disp('Forced state solution (from step input):');
pretty(x_forced_t)

disp('Total state solution (unforced + forced):');
pretty(x_total_t)

% Direct evaluation using subs()
x1_vals = double(subs(x_total_t(1), t, t_vals));
x2_vals = double(subs(x_total_t(2), t, t_vals));
y_vals = double(subs(y_total_t, t, t_vals));

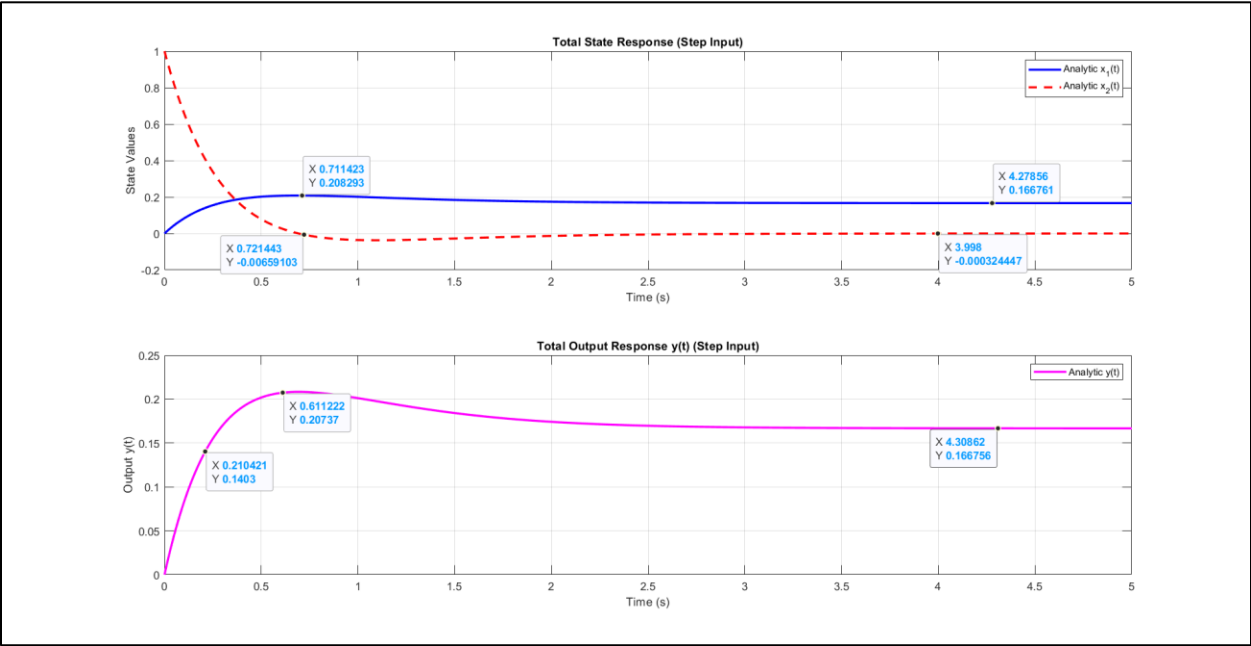
% Plot results
figure;

% State responses
subplot(2,1,1);
plot(t_vals, x1_vals, 'b', 'LineWidth', 2);
hold on;
plot(t_vals, x2_vals, 'r--', 'LineWidth', 2);
title('Total State Response (Step Input)');
xlabel('Time (s)');
ylabel('State Values');
legend('Analytic x_1(t)', 'Analytic x_2(t)');
grid on;

% Output response
subplot(2,1,2);
plot(t_vals, y_vals, 'm', 'LineWidth', 2);
hold on;
title('Total Output Response y(t) (Step Input)');
xlabel('Time (s)');
ylabel('Output y(t)');
legend('Analytic y(t)');
grid on;

disp(' ');
disp('Steady-State Values:');
disp(['x1(?) = ' char(ss_x1)]);
disp(['x2(?) = ' char(ss_x2)]);
disp(['y(?) = ' char(ss_y)]);
```

Output:





=== Forced Response Analysis ===

Forced state solution (from step input):

$$\frac{1}{6} \left( \frac{\exp(-3t) - \exp(-2t)}{1} \right)$$

Total state solution (unforced + forced):

$$\frac{1}{6} \left( \frac{\exp(-2t) - \exp(-3t)}{2} \right)$$

Steady-State Values:

$$x_1(\infty) = 1/6$$

$$x_2(\infty) = 0$$

$$y(\infty) = 1/6$$

Q8)

Code:

```
% Q8: State Feedback Design
disp('=== State Feedback Design ===');

% Original system step response
figure;
step(TF_builtin);
title('Original System Step Response');
grid on;

% Design specifications
zeta_desired = 0.7;      % Desired damping ratio
ts_desired = 1;         % Desired settling time (sec)

% Hand analysis to determine desired poles
wn = 4/(zeta_desired*ts_desired); % Natural frequency from settling time
sigma = zeta_desired*wn;      % Real part of poles
wd = wn*sqrt(1-zeta_desired^2); % Imaginary part

% Desired characteristic polynomial
desired_poly = (s + sigma + 1i*wd)*(s + sigma - 1i*wd);
desired_poly = expand(desired_poly);

% Convert to numerical polynomial
desired_coeffs = sym2poly(desired_poly);

% Hand calculation of K matrix
% Characteristic polynomial of A-BK: s^2 + (5+K2)s + (6+K1)
% Compare with desired polynomial: s^2 + 2*zeta*wn*s + wn^2

K1 = desired_coeffs(3) - 6; % From constant term
K2 = desired_coeffs(2) - 5; % From s term
K = [K1 K2];

disp('Desired closed-loop poles:');
disp([-sigma+1i*wd, -sigma-1i*wd]);

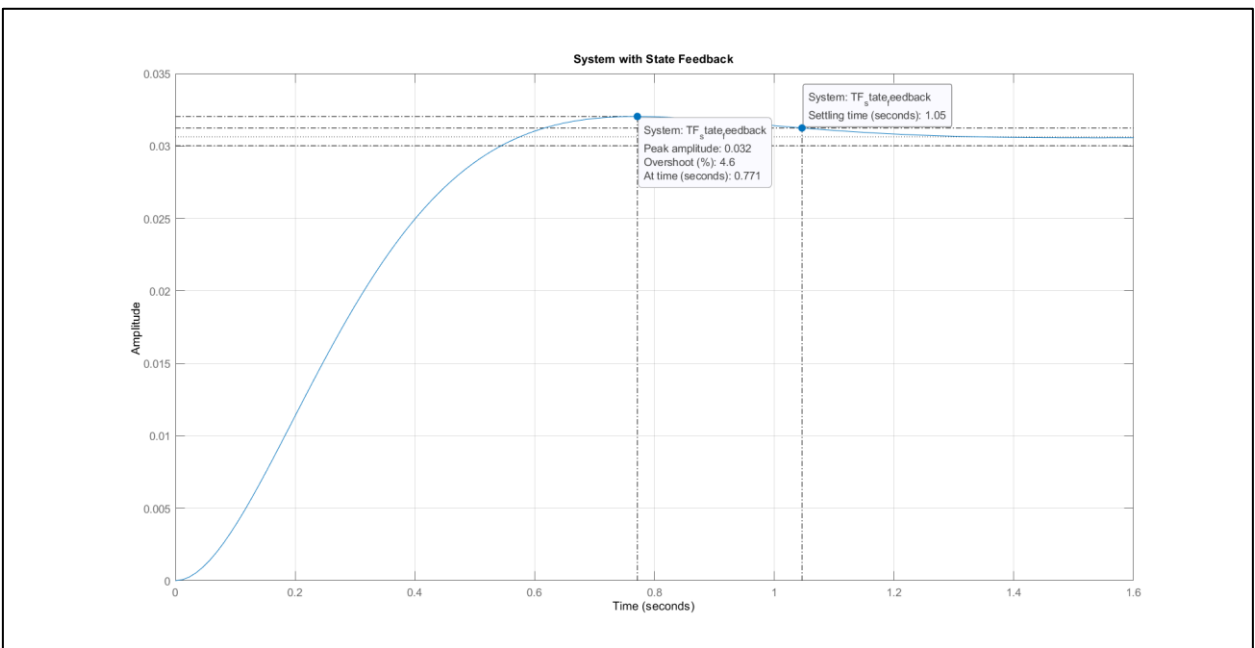
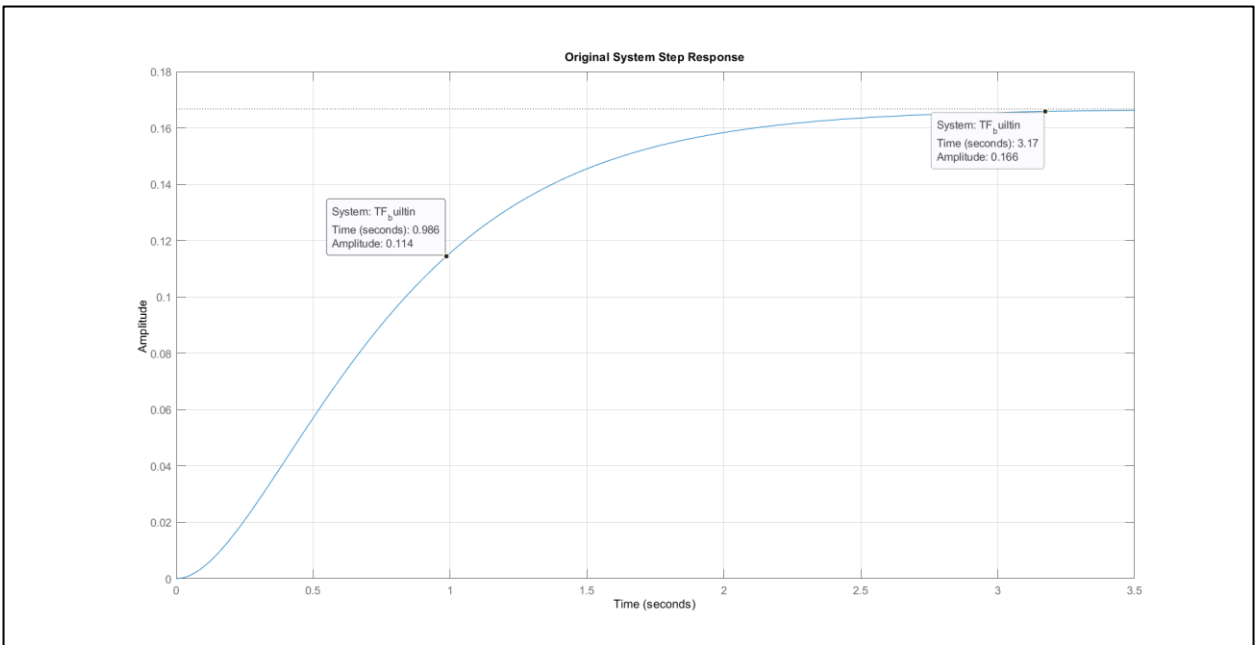
disp('Feedback gain matrix K:');
disp(K);

% Verification
Ac = A - B*K;
[num_2, denum_2] = ss2tf(Ac,B,C,D);
TF_state_feedback = tf(num_2, denum_2);

% Step response analysis
figure;
step_info = stepinfo(TF_state_feedback);
step(TF_state_feedback);
title('System with State Feedback');
grid on;

disp('Closed-loop system performance:');
disp(['Settling Time: ', num2str(step_info.SettlingTime), ' sec']);
disp(['Overshoot: ', num2str(step_info.Overshoot), '%']);
```

Output:



=== State Feedback Design ===

Desired closed-loop poles:

$$-4.0000 + 4.0808i \quad -4.0000 - 4.0808i$$

Feedback gain matrix K:

$$26.6531 \quad 3.0000$$

Closed-loop system performance:

Settling Time: 1.0463 sec

Overshoot: 4.5986%

## Appendix

### Code Plot Code:

```
clc;
clear;
close all;

%-----Q1----- Define G(s) and H(s)
% Define the open-loop transfer function G(s)
num_G = 1;
den_G = [1 1 0]; %  $s(s+1) = s^2 + s$ 

% Define the feedback transfer function H(s)
num_H = [1];
den_H = [1]; % Unity Feedback
[G_S, H_S] = create_system(num_G, den_G, num_H, den_H)

%-----Q2----- Step Response of G(s) (Open-Loop)
% Plot step response of G(s)
draw_step(G_S, 'Open-Loop System G(s)');

%-----Q3----- Closed-Loop Analysis
disp('Closed-Loop TF using feedback():');
T_feedback = feedback(G_S, H_S)

disp('Closed-Loop TF using manual formula (G/(1+GH)):');
T_manual = (1 / (1 + G_S * H_S)) * G_S; % Equivalent to  $T(s) = G/(1+GH)$ 
T_manual = minreal(T_manual) % Cancel common terms

%-----Q4----- Step Response of T(s) (Closed-Loop)
% Plot step response of T(s)
draw_step(T_feedback, 'Closed-Loop System T(s)');

%-----Q5----- locations of the poles
draw_poles(T_feedback);

%-----Q8----- Ramp Response
[ess, r_t_out, r_y_out] = draw_ramp(T_feedback, 700+200, 700);

%-----Q9----- Frequency Response
[Gm, Pm, Wgc, Wpc] = draw_Bode_Plot(G_S*H_S);

%-----Functions-----

function [G_S, H_S] = create_system(num_G, den_G, num_H, den_H)
% CREATE_SYSTEM Creates open-loop and feedback transfer functions
% [G_S, H_S] = create_system(num_G, den_G, num_H, den_H)
%
% Inputs:
% num_G - Numerator coefficients of G(s)
% den_G - Denominator coefficients of G(s)
% num_H - Numerator coefficients of H(s) (default: 1)
% den_H - Denominator coefficients of H(s) (default: 1)
%
% Outputs:
% G_S - Open-loop transfer function
% H_S - Feedback transfer function

% Set default unity feedback if not specified
if nargin < 3
```

```

        num_H = 1;
        den_H = 1;
    end

    % Create transfer functions
    G_S = tf(num_G, den_G)
    H_S = tf(num_H, den_H)
end

function [wn, zeta, response_info] = draw_step(sys, sys_name)
% DRAW_STEP Plots step response and returns key performance metrics
% [response_info] = draw_step(sys, sys_name)
%
% Inputs:
%     sys - Transfer function (tf object)
%     sys_name - Name of the system for title (string)
%
% Outputs:
%     response_info - Structure containing:
%         .poles - System poles
%         .stability - Stability classification
%         .peak_response - Peak response value and time
%         .settling_time - Time to settle within 2% of final value
%         .rise_time - 10-90% rise time
%         .steady_state - Final steady-state value
%     Figure with step response

% Create figure
figure;

% Get step response data
[y, t] = step(sys);

% Plot step response
step(sys);
title(['Step Response of ', sys_name]);
grid on;

% Calculate response characteristics
response_info = struct();
response_info.poles = pole(sys);

% Stability determination
if all(real(response_info.poles) < 0)
    response_info.stability = 'stable (all poles in LHP)';
elseif any(real(response_info.poles) > 0)
    response_info.stability = 'unstable (at least one pole in RHP)';
else
    response_info.stability = 'marginally stable (poles on imaginary
axis)';
end

% Peak response (overshoot)
[response_info.peak_response.value, peak_idx] = max(y);
response_info.peak_response.time = t(peak_idx);

% Steady-state value (last 10% of response)
steady_state_val = mean(y(end-round(length(y)*0.1):end));
response_info.steady_state = steady_state_val;

% Settling time (within 2% of steady-state)

```

```

    settled_idx = find(abs(y - steady_state_val) > 0.02*steady_state_val, 1,
'last');
    if isempty(settled_idx)
        response_info.settling_time = 0;
    else
        response_info.settling_time = t(settled_idx);
    end

    % Rise time (10% to 90% of steady-state)
    rise_start = find(y >= 0.1*steady_state_val, 1);
    rise_end = find(y >= 0.9*steady_state_val, 1);
    if ~isempty(rise_start) && ~isempty(rise_end)
        response_info.rise_time = t(rise_end) - t(rise_start);
    else
        response_info.rise_time = NaN;
    end

    % Display results in command window
    disp(['System: ', sys_name]);
    disp(['Poles: ', num2str(response_info.poles)]);
    disp(['Stability: ', response_info.stability]);
    disp(['Over shoot MP: ', num2str(100*(response_info.peak_response.value-
1)), ' ... ', num2str(response_info.peak_response.time), ' sec']);
    % Damping characteristics (for complex poles)
    if ~isreal(response_info.poles)
        [wn, zeta] = damp(sys);
        fprintf('Damping ratio (?): %.3f\n', zeta(1));
        fprintf('Natural frequency (?n): %.3f rad/s\n', wn(1));
    end

    disp(['Settling time (2%): ', num2str(response_info.settling_time), '
sec']);
    disp(['Rise time (10-90%): ', num2str(response_info.rise_time), ' sec']);
    disp(['Steady-state value: ', num2str(response_info.steady_state)]);
end

function [poles] = draw_poles(sys)
% DRAW_POLES Plots pole-zero map and returns system poles
% [poles] = draw_poles(sys)
%
% Input:
% sys - Transfer function (tf object) or state-space model
%
% Output:
% poles - Array of system poles
%
% Displays:
% - Pole-zero plot
% - Pole locations in command window
% - Stability information

% Create figure
figure;

% Plot pole-zero map
pzmap(sys);
title(['Pole-Zero Map of: ' inputname(1)]);
grid on;

% Get poles

```

```

poles = pole(sys);

% Display poles
disp(['Poles of ' inputname(1) ':']);
disp(poles);

% Damping characteristics (for complex poles)
if ~isreal(poles)
    [wn, zeta] = damp(sys);
    fprintf('Damping ratio (?): %.3f\n', zeta(1));
    fprintf('Natural frequency (?n): %.3f rad/s\n', wn(1));
end

end

function [ess, t_out, y_out] = draw_ramp(sys, t_end, zoom_time)
% DRAW_RAMP Plots ramp response in three subplots
% [ess, t_out, y_out] = draw_ramp(sys, t_end, zoom_time)
%
% Inputs:
% sys - Closed-loop transfer function (tf object)
% t_end - End time for simulation (default: 100 sec)
% zoom_time - Time to zoom in (default: 700 sec)
%
% Outputs:
% ess - Steady-state error
% t_out - Time vector
% y_out - System response vector
%
% Generates figure with three subplots:
% 1. Ideal ramp input
% 2. System response
% 3. Zoomed comparison at specified time
%
% Set defaults if not provided
if nargin < 2
    t_end = 100;
end
if nargin < 3
    zoom_time = 700;
end

% Create time vector
t = 0:0.1:t_end;

%getting the ramp
ramp = tf(1,[1 0]);

% Get response data
[y_sys, t_sys] = step(sys.*ramp, t);
[y_ideal, t_ideal] = step(ramp, t);

% Create figure with three subplots
figure;

% Subplot 1: Ideal ramp input
subplot(2,1,1);
plot(t_ideal, y_ideal, 'b');
hold on;
plot(t_sys, y_sys, 'r--');

```



```

title('Ramp Response');
xlabel('Time (sec)');
ylabel('Amplitude');
legend('Ideal', 'System', 'Location', 'northwest');
grid on;
hold off;

% Subplot 2: Zoomed comparison
subplot(2,1,2);
plot(t_ideal, y_ideal, 'b');
hold on;
plot(t_sys, y_sys, 'r--');
xlim([zoom_time-50 zoom_time+50]);
title(['Zoomed Comparison at t = ', num2str(zoom_time), ' sec']);
xlabel('Time (sec)');
ylabel('Amplitude');
legend('Ideal', 'System', 'Location', 'northwest');
grid on;
hold off;

% Calculate steady-state error (use last 10% of simulation)
final_idx = round(0.9*length(t_sys)):length(t_sys);
ess = mean(y_ideal(final_idx) - y_sys(final_idx));

% Display results
disp(['Steady-state error (ess): ', num2str(ess)]);

% Return output data if requested
if nargin > 1
    t_out = t_sys;
    y_out = y_sys;
end
end

function [Gm, Pm, Wgc, Wpc] = draw_Bode_Plot(sys)
% BODE_PLOT Analyzes system stability margins and compares margin()
% Bode_Plot(sys)
%
% Input:
% sys - Transfer function (tf object or state-space model)
% Outputs:
% Gm - Gain margin (dB)
% Pm - Phase margin (degrees)
% Wgc - Gain crossover frequency (rad/sec)
% Wpc - Phase crossover frequency (rad/sec)

% Create margin plot
figure;
margin(sys);
grid on;

% Get stability margins
[Gm, Pm, Wgc, Wpc] = margin(sys);

% Display results
disp(['=== Stability Margins for ' inputname(1) ' ===']);
disp(['Gain Margin: ', num2str(Gm), ' dB at ', num2str(Wgc), ' rad/s']);
disp(['Phase Margin: ', num2str(Pm), '° at ', num2str(Wpc), ' rad/s']);
end

```

## State Space Code::

```
clc
clear all
close all

% Given system matrices
A = [0 1; -6 -5];
B = [0; 1];
C = [1 0];
D = [0];
n = 2; % System order
sys = ss(A,B,C,D); % State Space model
x0 = [0; 1]; % Initial condition

% Q2: Transfer function conversion
[num, den] = ss2tf(A,B,C,D);
syms s
TF_Manual = C*inv(s*eye(n)-A)*B + D
TF_builtin = tf(num,den)

% Q3: State transition matrix calculation
% Compute  $\Phi(s) = [sI - A]^{-1}$ 
Phi_s = inv(s*eye(n) - A);

% Compute  $\Phi(t)$  by inverse Laplace transform
syms t
Phi_t = ilaplace(Phi_s);

% Verify  $\Phi(0) = I$ 
Phi_0 = subs(Phi_t, t, 0);

% Display results
disp('State transition matrix in s-domain  $\Phi(s)$ :');
pretty(Phi_s)

disp('State transition matrix in time domain  $\Phi(t)$ :');
pretty(Phi_t)

disp('Verification of  $\Phi(0) = I$ :');
disp(Phi_0);

% Q4: Verify that  $\dot{\Phi}(t) = A\Phi(t)$ 
Phi_dot = diff(Phi_t, t); % Take time derivative of  $\Phi(t)$ 
A_Phi = A*Phi_t; % Multiply A with  $\Phi(t)$ 

disp('Time derivative of state transition matrix  $\dot{\Phi}(t)$ :');
pretty(Phi_dot)

disp('A* $\Phi(t)$ :');
pretty(A_Phi)

disp('Verification successful:  $\dot{\Phi}(t) = A\Phi(t)$ ');

% Q5 Check Controllability and Observability
% Check Controllability
Co = ctrb(A, B); % Controllability matrix
rank_Co = rank(Co);
```

```

disp('Controllability Matrix:');
disp(Co);
disp(['Rank of Controllability Matrix: ', num2str(rank_Co)]);

if rank_Co == n
    disp('System is Controllable (as expected)');
else
    disp('System is Not Controllable (unexpected for this system)');
end

% Check Observability
Ob = obsv(A, C); % Observability matrix
rank_Ob = rank(Ob);
disp('Observability Matrix:');
disp(Ob);
disp(['Rank of Observability Matrix: ', num2str(rank_Ob)]);

if rank_Ob == n
    disp('System is Observable (as expected)');
else
    disp('System is Not Observable (unexpected for this system)');
end

% Q6: Unforced (Homogeneous) Response
disp('=== Unforced Response Analysis ===');

% Compute state solution  $x(t) = \Phi(t)x_0$ 
x_t = Phi_t * x0;

disp('Unforced state solution  $x(t)$ :');
pretty(x_t)

% Compute output solution  $y(t) = Cx(t) + Du(t)$ 
% Since  $u(t)=0$  for unforced response:
y_t = C*x_t + D*0;

disp('Unforced output response  $y(t)$ :');
pretty(y_t)

% Plot the results
t_vals = linspace(0, 5, 500); % Time vector from 0 to 5 seconds

% Convert symbolic expressions to numeric functions
x1_func = matlabFunction(x_t(1));
x2_func = matlabFunction(x_t(2));
y_func = matlabFunction(y_t);

% Evaluate solutions
x1_vals = arrayfun(x1_func, t_vals);
x2_vals = arrayfun(x2_func, t_vals);
y_vals = arrayfun(y_func, t_vals);

% Plot state responses
figure;
subplot(2,1,1);
plot(t_vals, x1_vals, 'b', 'LineWidth', 2);
hold on;
plot(t_vals, x2_vals, 'r--', 'LineWidth', 2);
title('Unforced State Response');
xlabel('Time (s)');
ylabel('State Values');

```

```

legend('x_1(t)', 'x_2(t)');
grid on;

% Plot output response
subplot(2,1,2);
plot(t_vals, y_vals, 'm', 'LineWidth', 2);
title('Unforced Output Response y(t)');
xlabel('Time (s)');
ylabel('Output y(t)');
grid on;

% Compare with MATLAB's built-in initial() function
[~,t_num,x_num] = initial(sys,x0,t_vals(end));
y_num = x_num*C'; % Equivalent to C*x since D=0

% Display symbolic solutions
disp(' ');
disp('Analytic Solutions:');
disp('x1(t) = '); pretty(x_t(1))
disp('x2(t) = '); pretty(x_t(2))
disp('y(t) = '); pretty(y_t)

% Q7: Forced Response Analysis (Unit Step Input)
disp('=== Forced Response Analysis ===');

% Using Frequency Domain Approach
U_s = 1/s; % Laplace transform of unit step
U_t = ilaplace(U_s);

% Compute forced component in frequency domain
X_forced_s = Phi_s * B * U_s;

% Convert to time domain
x_forced_t = ilaplace(X_forced_s);

% Total solution (homogeneous + forced)
x_total_t = x_t + x_forced_t;

% Output solution
y_total_t = C*x_total_t + D*U_t; % D*u(t) where u(t)=1 for t>0

disp('Forced state solution (from step input):');
pretty(x_forced_t)

disp('Total state solution (unforced + forced):');
pretty(x_total_t)

% Direct evaluation using subs()
x1_vals = double(subs(x_total_t(1), t, t_vals));
x2_vals = double(subs(x_total_t(2), t, t_vals));
y_vals = double(subs(y_total_t, t, t_vals));

% Plot results
figure;

% State responses
subplot(2,1,1);
plot(t_vals, x1_vals, 'b', 'LineWidth', 2);
hold on;
plot(t_vals, x2_vals, 'r--', 'LineWidth', 2);
title('Total State Response (Step Input)');

```

```

xlabel('Time (s)');
ylabel('State Values');
legend('Analytic x_1(t)', 'Analytic x_2(t)');
grid on;

% Output response
subplot(2,1,2);
plot(t_vals, y_vals, 'm', 'LineWidth', 2);
hold on;
title('Total Output Response y(t) (Step Input)');
xlabel('Time (s)');
ylabel('Output y(t)');
legend('Analytic y(t)');
grid on;

% Display final steady-state values
ss_x1 = limit(x_total_t(1), t, inf);
ss_x2 = limit(x_total_t(2), t, inf);
ss_y = limit(y_total_t, t, inf);

disp(' ');
disp('Steady-State Values:');
disp(['x1(?) = ' char(ss_x1)]);
disp(['x2(?) = ' char(ss_x2)]);
disp(['y(?) = ' char(ss_y)]);

% Q8: State Feedback Design
disp('=== State Feedback Design ===');

% Original system step response
figure;
step(TF_builtin);
title('Original System Step Response');
grid on;

% Design specifications
zeta_desired = 0.7; % Desired damping ratio
ts_desired = 1; % Desired settling time (sec)

% Hand analysis to determine desired poles
wn = 4/(zeta_desired*ts_desired); % Natural frequency from settling time
sigma = zeta_desired*wn; % Real part of poles
wd = wn*sqrt(1-zeta_desired^2); % Imaginary part

% Desired characteristic polynomial
desired_poly = (s + sigma + 1i*wd)*(s + sigma - 1i*wd);
desired_poly = expand(desired_poly);

% Convert to numerical polynomial
desired_coeffs = sym2poly(desired_poly);

% Hand calculation of K matrix
% Characteristic polynomial of A-BK: s^2 + (5+K2)s + (6+K1)
% Compare with desired polynomial: s^2 + 2*zeta*wn*s + wn^2

K1 = desired_coeffs(3) - 6; % From constant term
K2 = desired_coeffs(2) - 5; % From s term
K = [K1 K2];

disp('Desired closed-loop poles:');

```

```

disp([-sigma+1i*wd, -sigma-1i*wd]);

disp('Feedback gain matrix K:');
disp(K);

% Verification
Ac = A - B*K;
[num_2, denum_2] = ss2tf(Ac,B,C,D);
TF_state_feedback = tf(num_2, denum_2);

% Step response analysis
figure;
step_info = stepinfo(TF_state_feedback);
step(TF_state_feedback);
title('System with State Feedback');
grid on;

disp('Closed-loop system performance:');
disp(['Settling Time: ', num2str(step_info.SettlingTime), ' sec']);
disp(['Overshoot: ', num2str(step_info.Overshoot), '%']);

```