

# Thermal Power Plants

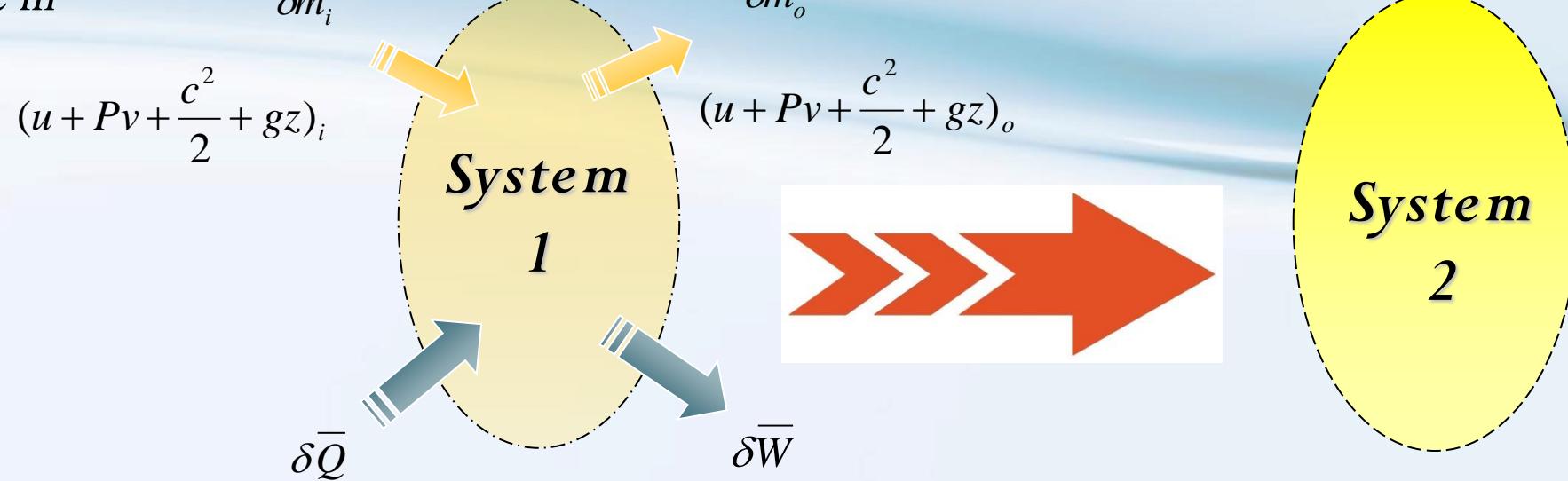
Section-1

# -Fundamentals of thermodynamics principles:

## 1. First law of thermodynamics:

*Energy can neither be created nor destroyed; it can only be transformed from one form to another*

Inlet Energy - Outlet Energy = Final energy of system - Initial energy of system



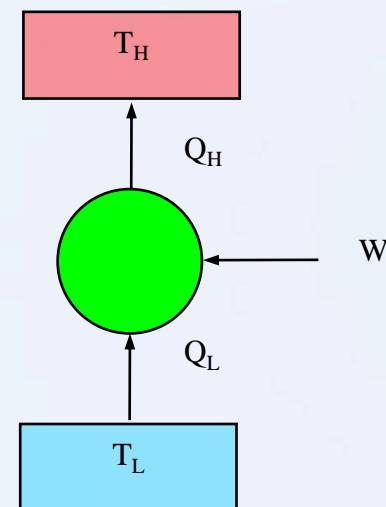
When system at rest:  $Q_{12} - W_{12} = u_2 - u_1 \quad J/kg$

## 2. Second law of thermodynamics:

*“It is impossible to convert heat completely to work; a part of it must be rejected to a heat sink at a lower temperature”*

- For reversible heat engine:  $\frac{Q_H}{Q_L} = \frac{T_H}{T_L}$

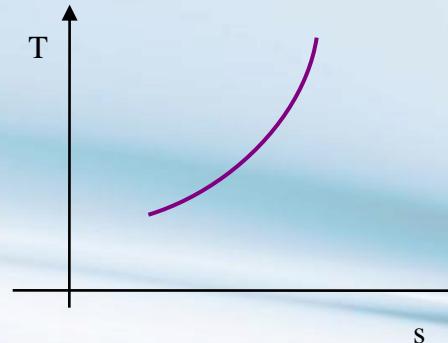
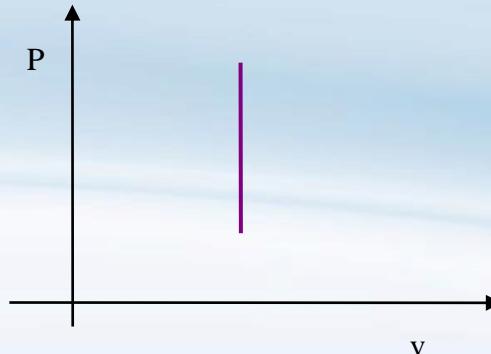
- For actual heat engine:



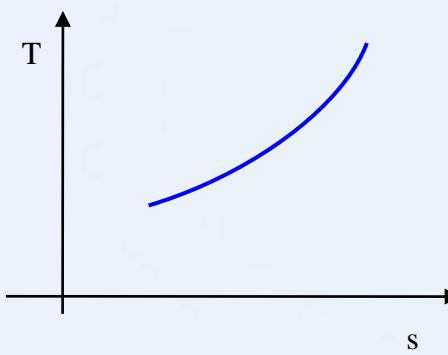
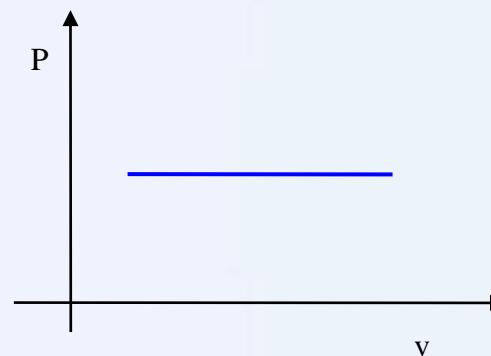
$$\eta = \frac{W}{Q_H}$$

## 2. Thermodynamic processes:

A-Constant volume process (isochoric or isometric)

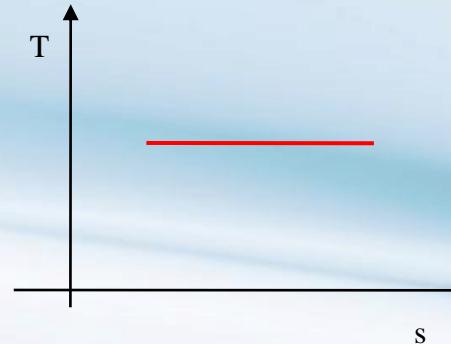
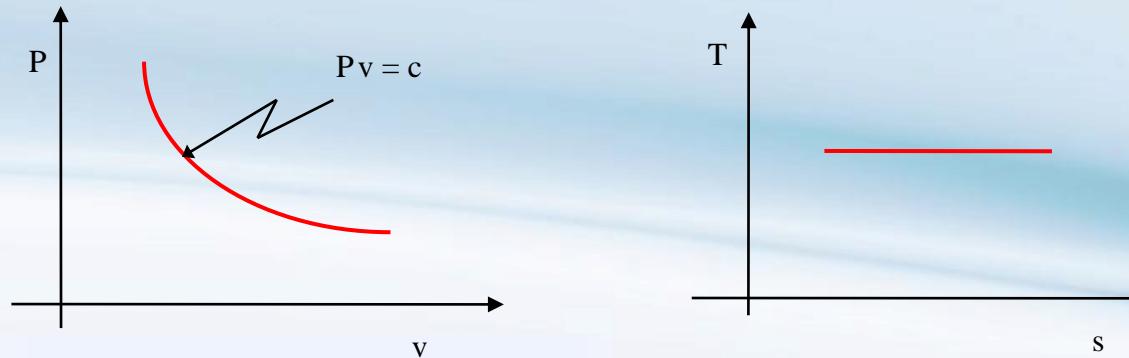


B-Constant pressure process (isobaric)

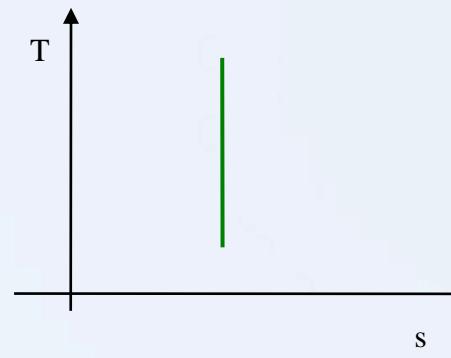
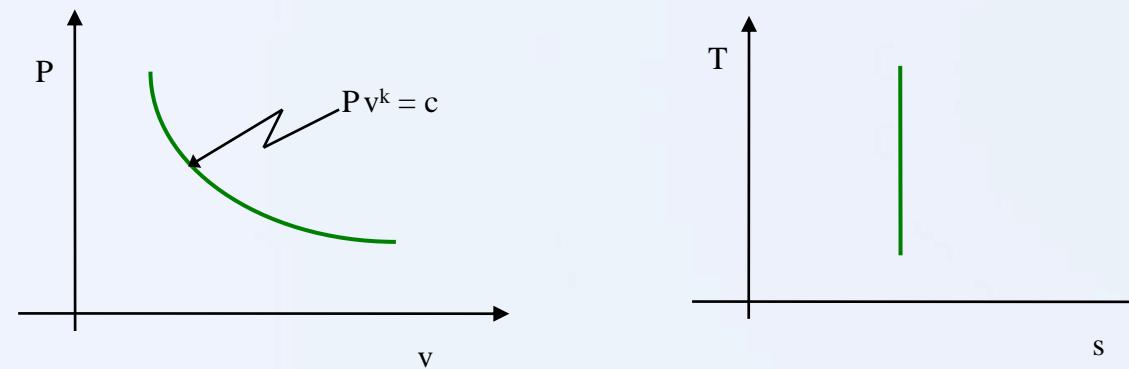


## 2. Thermodynamic processes:

C-Constant temperature process (isothermal)



D-Constant entropy process (isentropic)



## 2. Thermodynamic processes:

■ for  $n \neq 1$

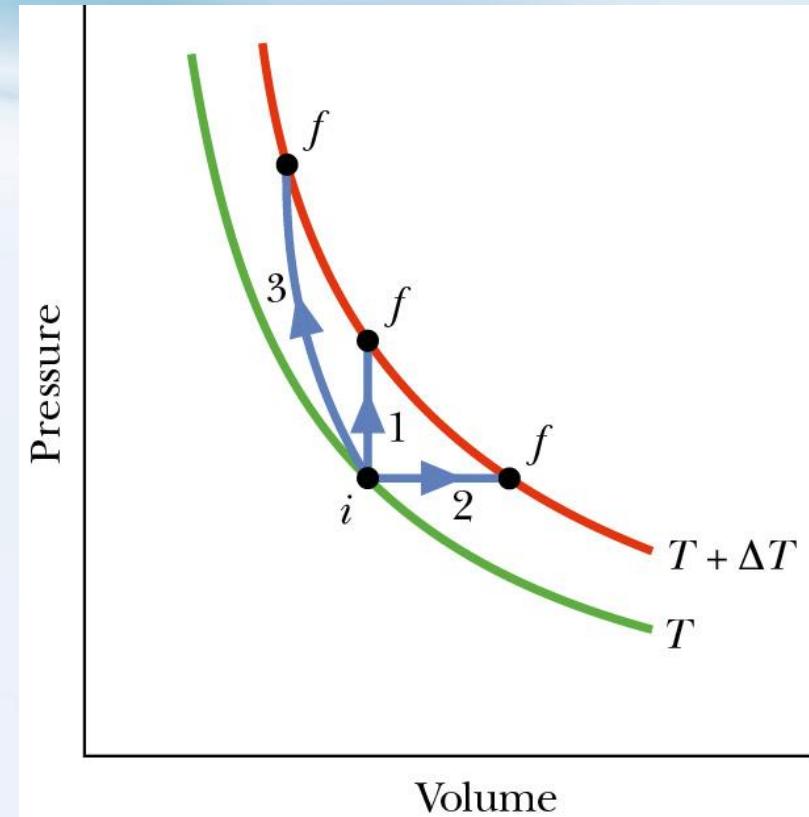
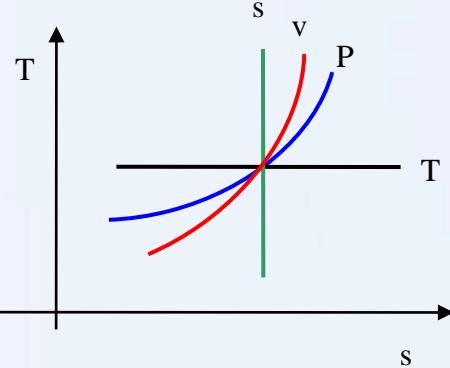
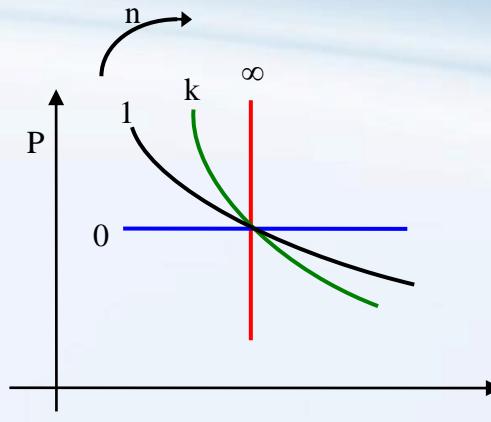


$$W = \frac{P_1 v_1 - P_2 v_2}{n-1} \quad J/kg$$

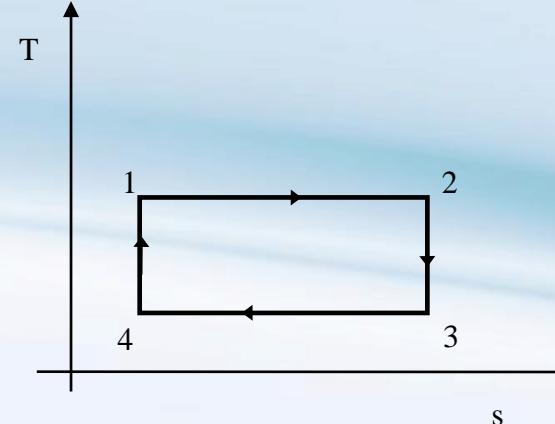
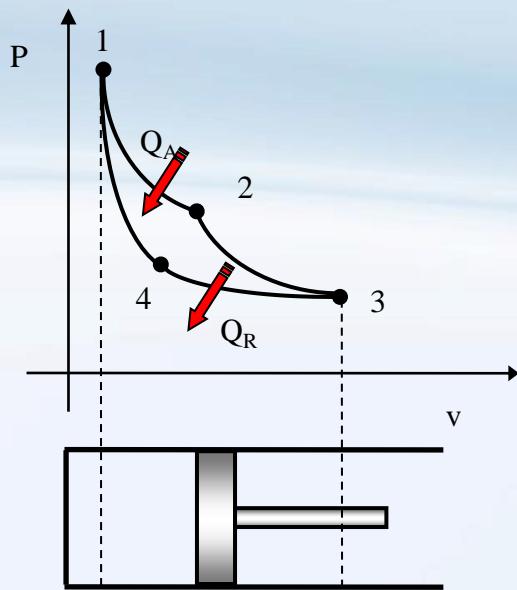
■ for  $n = 1$



$$W = P v \ln \frac{v_2}{v_1} \quad J/kg$$



- In a **Carnot cycle** the working substance is subjected to a cyclic operation consisting of two isothermal and two isentropic ( adiabatic + reversible ) processes.



- 1-2 isothermal expansion [ **heat addition** ]
- 2-3 isentropic expansion
- 3-4 isothermal compression [ **heat rejection** ]
- 4-1 isentropic compression

# Coefficient of Performance

■ *For heat pump :*

$$COP_p = \beta_p = \frac{Q_H}{W} = \frac{Q_H}{Q_H - Q_L}$$

□ *for reversed Carnot heat pump :*

$$COP_{p_C} = \beta_{p_C} = \frac{T_H}{T_H - T_L}$$

---

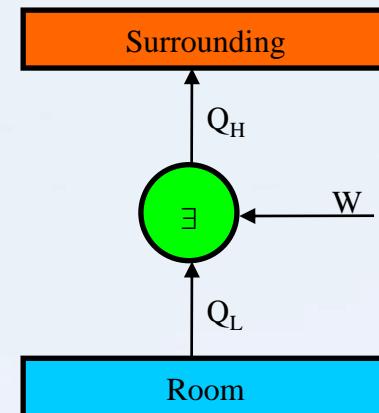
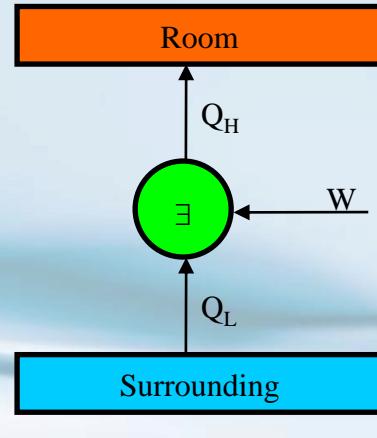
---

■ *For refrigerator :*

$$COP_R = \beta_R = \frac{Q_L}{W} = \frac{Q_L}{Q_H - Q_L}$$

□ *for reversed Carnot refrigerator :*

$$COP_{R_C} = \beta_{R_C} = \frac{T_L}{T_H - T_L}$$

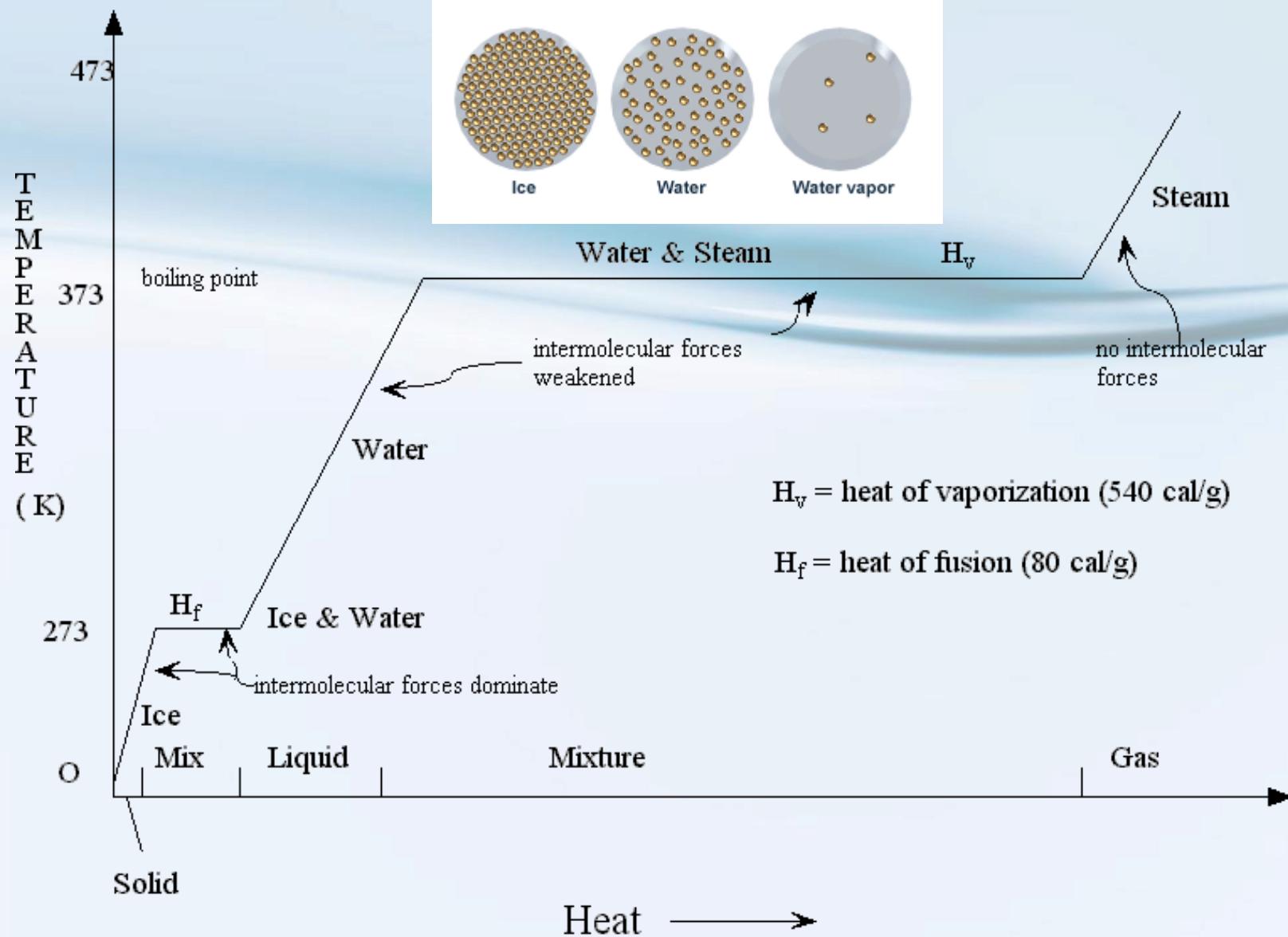


## Ideal Gas Formulas

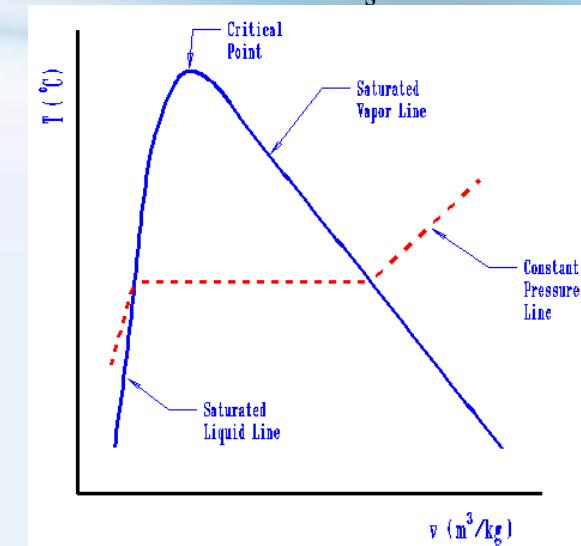
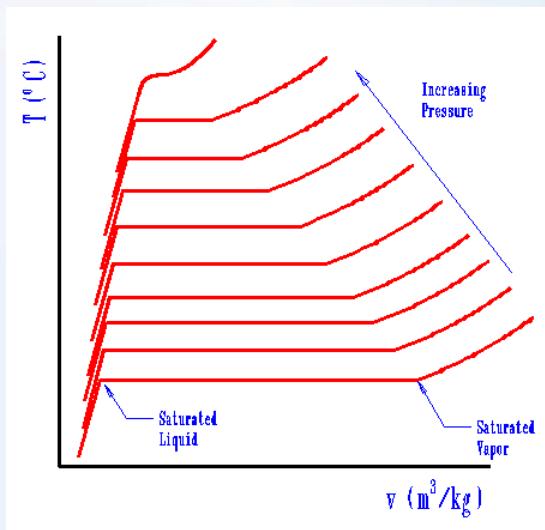
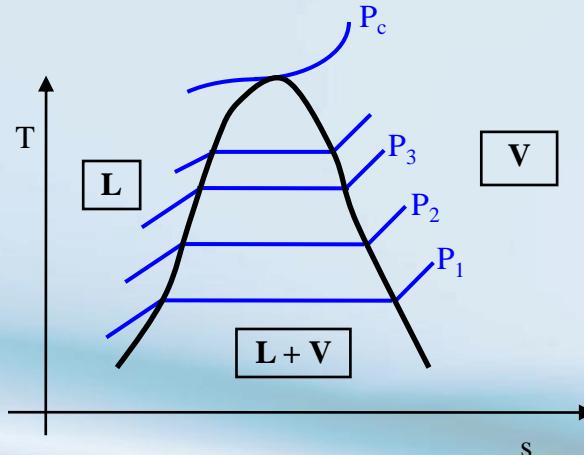
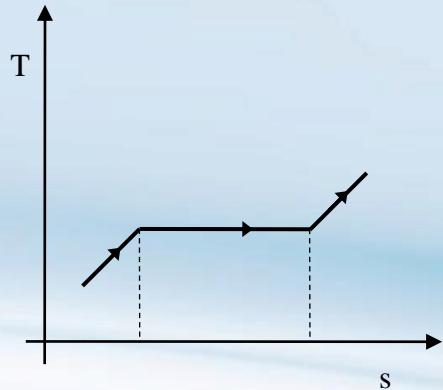
Process	Constant Volume	Constant Pressure	Isothermal	Isentropic	Polytropic
<b>n</b>	$\infty$	<b>0</b>	<b>1</b>	<b>k</b>	<b><math>0 \leq n \leq k</math></b>
P,v,T relations	$\frac{P_2}{P_1} = \frac{T_2}{T_1}$	$\frac{v_2}{v_1} = \frac{T_2}{T_1}$	$P_1 v_1 = P_2 v_2$	$\frac{P_2}{P_1} = \left(\frac{v_1}{v_2}\right)^k = \left(\frac{T_2}{T_1}\right)^{\frac{k}{k-1}}$	$\frac{P_2}{P_1} = \left(\frac{v_1}{v_2}\right)^n = \left(\frac{T_2}{T_1}\right)^{\frac{n}{n-1}}$
$\int P dv$	0	$P(v_2 - v_1)$	$Pv \ln \frac{v_2}{v_1}$	$\frac{P_1 v_1 - P_2 v_2}{k-1}$	$\frac{P_1 v_1 - P_2 v_2}{n-1}$
$-\int v dP$	$v(P_1 - P_2)$	0	$Pv \ln \frac{P_1}{P_2}$	$\frac{k(P_1 v_1 - P_2 v_2)}{k-1}$	$\frac{n(P_1 v_1 - P_2 v_2)}{n-1}$
$u_2 - u_1$	$c_v(T_2 - T_1)$	$c_v(T_2 - T_1)$	0	$c_v(T_2 - T_1)$	$c_v(T_2 - T_1)$
$h_2 - h_1$	$c_p(T_2 - T_1)$	$c_p(T_2 - T_1)$	0	$c_p(T_2 - T_1)$	$c_p(T_2 - T_1)$
Q	$c_v(T_2 - T_1)$	$c_p(T_2 - T_1)$	$Pv \ln \frac{v_2}{v_1}$	0	$c_n(T_2 - T_1)$
Specific Heat	$c_v$	$c_p$	$\infty$	0	$c_n = \frac{c_v(k-n)}{1-n}$
$s_2 - s_1$	$c_v \ln \frac{T_2}{T_1}$	$c_p \ln \frac{T_2}{T_1}$	$R \ln \frac{v_2}{v_1}$	0	$c_n \ln \frac{T_2}{T_1}$

# Pure substance:

Phase Change Diagram for Water



# Saturation lines:



There are two saturation lines; saturated liquid line (SLL) and saturated vapor line (SVL). Between these saturation lines there is a mixture of liquid and vapor. In the mixed-phase regions, temperature and pressure cannot be specified independently.

# Triple point:

- The phase diagram is P-T diagram. It consists of three saturation lines; solid-liquid line, liquid-gas line and solid-gas line.

