### 1 Stress, Strain

$$\begin{split} \sigma &= \frac{F}{A} \\ \epsilon &= \frac{\Delta l}{L} \\ \sigma &= E \epsilon \\ \frac{F}{A} &= \frac{E \cdot \Delta l}{L}, F = \frac{AE}{L} \cdot \Delta l = k \Delta l \\ k &= \frac{AE}{L} \\ W &= \frac{1}{2} k \Delta x^2 = \frac{1}{2} F \Delta x = \frac{1}{2} \sigma A \epsilon L = \frac{\sigma^2 V_0}{2E} = \frac{\sigma \epsilon V_0}{2} \\ U &= \int \sigma d \epsilon \\ W &= U \cdot V_0 \rightarrow U = \frac{W}{V_0} \\ \epsilon_{th} &= \alpha \Delta T \end{split}$$

## 2 Equilibrium

$$\Sigma F=0,\;\Sigma M=0,\;if\;a=0,\;where\;F=ma,\;and\;M=Fd$$

## 3 Bridges

$$\begin{split} T_{supY} &= \frac{WL}{2} \\ T_{supX} &= \frac{WL^2}{8h} \\ T_{supMax} &= \sqrt{(\frac{WL}{2})^2 + (\frac{WL^2}{8h})^2}, \end{split}$$

Given loads placed equidistantly over the span

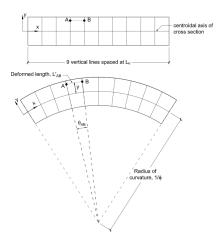
### 4 Rotation

#### 4.1 Motion

$$I_m=my^2, \ for \ a \ single \ point \ mass$$
 
$$I_m=\int_M y^2 dm = \rho \int_A y^2 dA$$

$$\begin{split} I &= \int_A y^2 dA \\ I &= \frac{bh^3}{12}, \ for \ a \ rectangle \ (comes \ from \ integrating) \\ F &= ma = my \\ M &= F_x y = my^2 \cdot \alpha = I_m \alpha \\ \omega &= \frac{d^2\theta}{dt^2} \\ \theta &= \omega_i t + \frac{1}{2}\alpha t^2 \\ a &= \alpha y \\ M &= Fy = m\alpha y^2 = I_m \alpha \end{split}$$

#### 4.2 Bending



$$\begin{split} \phi &= \frac{d\theta}{dx} \\ L'_{AB} &= \phi L_0 \cdot (y + \frac{1}{\phi}) = \phi y L_0 + L_0 \\ \epsilon(y) &= \frac{\Delta l}{L_0} = \frac{L'_{AB} - L_0}{L_0} = \phi y, \ \epsilon \ at \ the \ centroidal \ axis \ is \ 0 \\ \sigma &= E\epsilon \to \sigma(y) = E\phi y \\ \sigma &= F/A \to \Delta F = \sigma(y) \Delta A \\ M &= Fd \to \Delta M = \Delta Fy = \phi Ey^2 \Delta A \\ M &= \int_A \phi Ey^2 dA \end{split}$$

$$= \phi EI$$

### 5 Harmonic Motion

$$Pendulum: T = 2\pi\sqrt{\frac{l}{g}}$$

Mass Spring: 
$$2\pi\sqrt{\frac{m}{k}}$$

$$f = \frac{1}{T}$$

$$F = -kx$$

$$\omega = \frac{2\pi}{T}$$

$$a = -\omega^2 x$$

$$x(t) = A \cdot \sin(\omega t + \phi)$$

$$v = \omega A \cdot \cos(\omega t + \phi)$$

$$v = \pm \omega \sqrt{x_0^2 - x^2}$$

$$\frac{d^2x(t)}{dt^2} = -A\omega^2 \sin(\omega_n t + \phi)$$

$$E_k = \frac{1}{2} m\omega^2 (x_0^2 - x^2)$$

$$E_t = \frac{1}{2}m\omega^2 x^2$$

$$\tfrac{1}{2}mv^2=\tfrac{1}{2}kx^2\to v=\sqrt{x\tfrac{k}{m}}$$

#### Gravity

$$m\frac{d^2x(t)}{dt^2} + kx(t) = mg$$

$$x(t) = A\sin(\omega_n t + \phi) + \Delta_0$$

$$k = \frac{mg}{\Delta_0}$$

$$f_n = \frac{1}{2\pi} \sqrt{\frac{mg}{\Delta_0} \cdot \frac{1}{m}} = \frac{1}{2\pi} \sqrt{\frac{g}{\Delta_0}}$$

# 6 Staticaly Determined Structures

	Permited	Restrained	
Name	Degrees of	Degrees of	Reactions
	Freedom	Freedom	
Roller	$\Delta(x \oplus y), \ \theta$	$\Delta y = 0$	$F_y$
Pin	$\theta$	$\Delta x = \Delta y = 0$	$F_x, F_y$
Fixed End	N/A	$\Delta x = \Delta y = \Delta \theta = 0$	$F_x, F_y, M_{xy}$

# 7 Truss Brigde



- 1. Select Geometry
- 2. Determine Loads
  - Gravity
  - Wind
- 3. Analyze forces in members

- 4. Design components
- 5. Determine stiffeness
- 6. Check dynamic forces
- 7. Iterate

$$P_{central\ joint} = W \cdot \frac{s \cdot W_D}{2}$$

$$P_{end\ joint} = W \cdot \tfrac{s \cdot W_D}{4}$$