

1 Part D Background and Theory

a. Symbol Directory:

Experimental Quantities:		
d	Pipe Diameter	m
L	Length of Tube across which Head Loss is Measured	m
Q	Volumetric Flow Rate	$\frac{m^3}{s}$
h_1	Upstream Head	m
h_2	Downstream Head	m
T	Fluid Temperature	$^{\circ}C$

Computed Quantities:		
V_{av}	Average Fluid Velocity	$\frac{m}{s}$
$[Re]$	Reynold's Number	
h_K	Empirical Minor Head Loss	m
f	Friction Factor	
K	Minor Head Loss (Accessory) Coefficient	
L_{eq}	Equivalent Tube Length	m

Constants and Thermophysical Properties:		
g	Acceleration due to Gravity	$9.807 \frac{m}{s^2}$
ρ	Fluid Density	$\frac{kg}{m^3}$
μ	Fluid Dynamic Viscosity	$\frac{N}{m^2 s}$
ν	Fluid Kinematic Viscosity	$\frac{m^2}{s}$
ε	Surface Roughness of Pipe	m
$\varepsilon_s = \varepsilon_{smooth,ideal}$	Surface Roughness of Ideal Smooth Pipe	$0m$

**All non-standard values extracted from relevant sections of assignment.
Values not provided were either not needed or were determined as
a function of some other variable.*

b. Basic Calculations:

Average Velocity Assuming an incompressible fluid, for flow through a given tube, the average velocity, V_{av} at any cross-section can be determined from the volumetric flow rate, Q , anywhere in the stream and the diameter of the tube, d at the cross-section of interest as:

$$V_{av} = \frac{Q}{A_c} = \frac{4Q}{\pi d^2} \quad (1)$$

Reynold's Number As given in equation 8 of the assignment, the Reynold's Number can be determined by:

$$[Re] = \frac{\rho V_{av} d}{\mu} = \frac{V_{av} d}{\nu} \quad (2)$$

Head Loss By definition, head loss can be computed as:

$$h_k = h_1 - h_2 \quad (3)$$

c. System Calculations:

Head Loss and Flow Rate Measurement: If the head loss over a certain very short region which experiences a change in cross sectional area is known, the Bernoulli and Venturi principles can be applied to measure the volumetric flow rate through that area as outline in equation 11 of the assignment:

$$Q = C_d A_0 \sqrt{\frac{2g(\Delta h)}{1 - \left(\frac{A_0}{A_1}\right)^2}} = \frac{C_d \pi d_0^2}{4} \sqrt{\frac{2g(\Delta h)}{1 - \left(\frac{d_0}{d_1}\right)^4}} \quad (4)$$

where C_d is the discharge coefficient, h is the head loss, and A_0 and A_1 are the cross sectional areas of the inlet and outlet respectively.

Uncertainties: Per the specific request of prompt D.2, the uncertainty in calculated flow rates, Q , due to uncertainties in the measurements of d_0 and d_1 , can be determined by combining partial uncertainties of the equation for Q given in equation 11 of the assignment as follows:

$$\begin{aligned} \delta Q^2 &= \left[\frac{\partial Q}{\partial d_0} \right]^2 \delta d_0^2 + \left[\frac{\partial Q}{\partial d_1} \right]^2 \delta d_1^2 \\ \delta Q &= \sqrt{\left(\frac{\pi C_d d_0}{2} \sqrt{\frac{2(\Delta h)g}{1 - \frac{d_0^4}{d_1^4}}} + \frac{\pi \sqrt{2} C_d (\Delta h) d_0^5 g}{2 d_1^4 \left(\frac{d_0^4}{d_1^4} - 1 \right)^2 \sqrt{\frac{(\Delta h)g}{1 - \frac{d_0^4}{d_1^4}}}} \right)^2 \delta d_0^2 - \left(\frac{C_d^2 (\Delta h) d_0^{12} g \pi^2}{2 d_1^{10} \left(\frac{d_0^4}{d_1^4} - 1 \right)^3} \right) \delta d_1^2} \\ &\rightarrow \delta Q = \frac{\pi \sqrt{2}}{2} \sqrt{-\frac{C_d^2 (\Delta h) d_0^2 d_1^2 g (d_0^{10} \delta d_1^2 + d_1^{10} \delta d_0^2)}{(d_0^4 - d_1^4)^3}} \quad (5) \end{aligned}$$

where it is given that $\delta d_0 = \delta d_1 = 0.2mm = 2 \times 10^{-4}m$ for both the Orifice and Venturi tube diameter measurements.