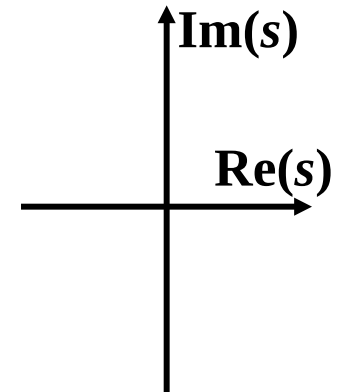
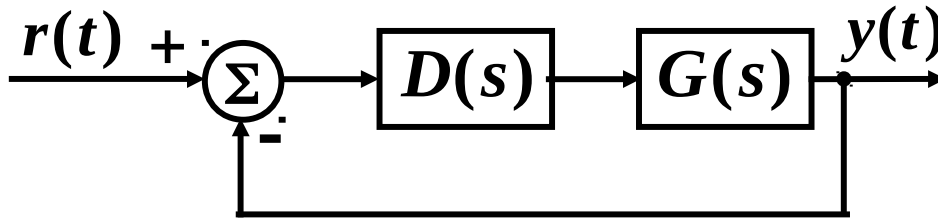


Frequency Response Methods

- **Frequency response methods**
 - **Advantages**
 - **Continuous-time systems (review)**
 - **Discrete-time systems**
 - **Position error constants**
 - **Phase lag due to sampling**
 - **Gain and phase margins**

Frequency Response Methods

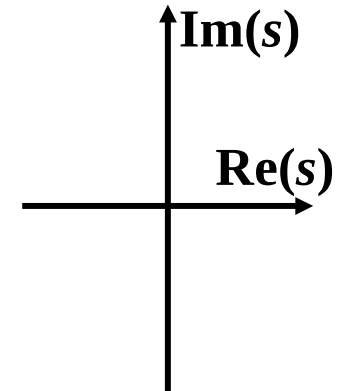
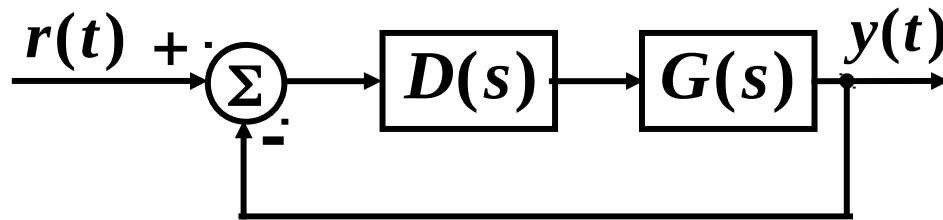


$$G(s) = K \frac{(s + z_1)(s + z_2) \cdots (s + z_m)}{(s + p_1)(s + p_2) \cdots (s + p_n)}, \quad m \leq n$$

$$G(s) = K_0 \frac{(s/z_1 + 1)(s/z_2 + 1) \cdots (s/z_m + 1)}{(s/p_1 + 1)(s/p_2 + 1) \cdots (s/p_n + 1)}$$

- Bode plots: using gain margin (GM) and phase margin (PM) to design compensators
- Nyquist plots: for stability analysis

Advantages of Frequency Response Methods

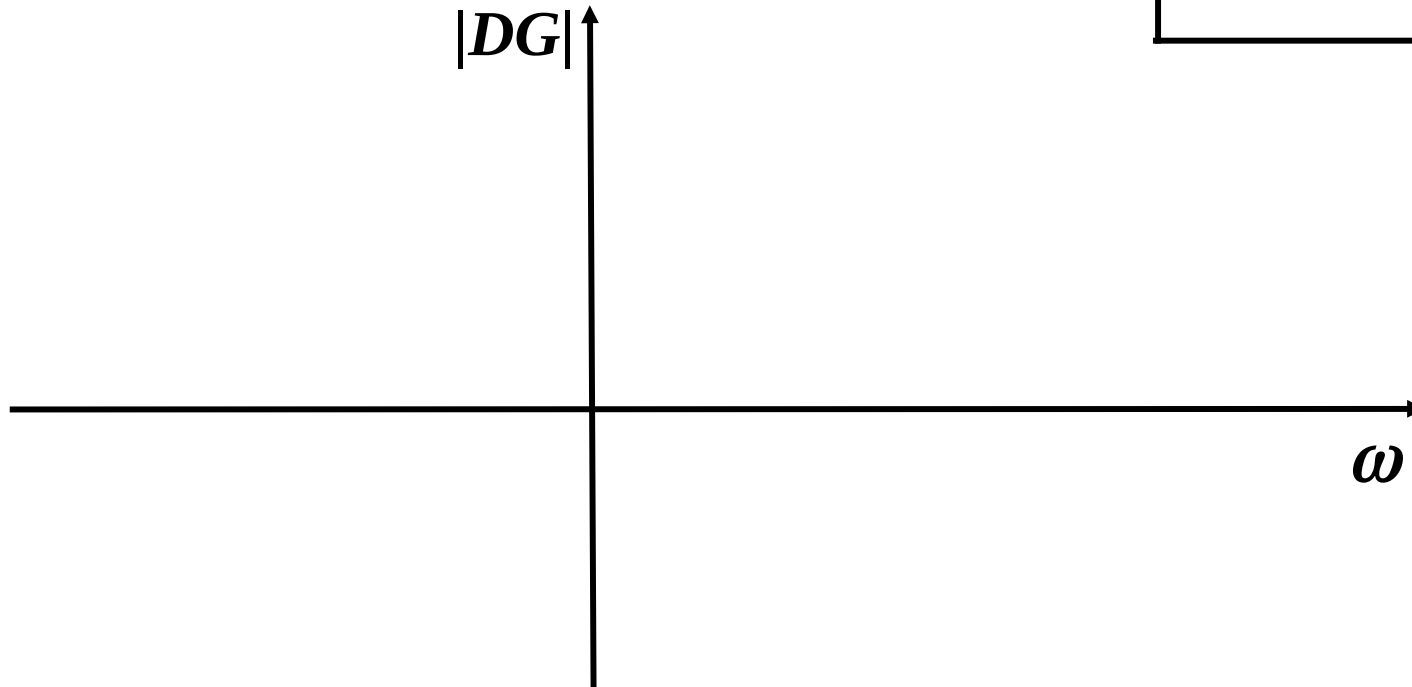
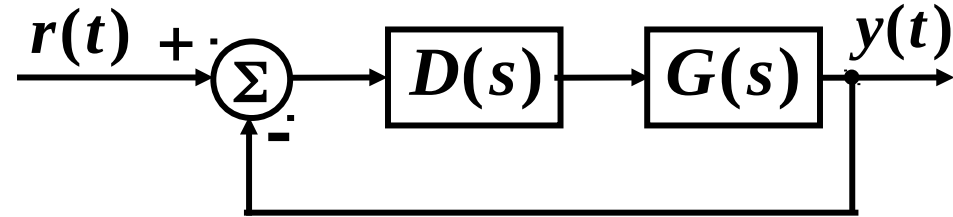


$$|G(s)| \Big|_{s=j\omega}$$

$$\angle G(s) \Big|_{s=j\omega}$$

1. Sketching Bode gain/phase plots is easy.
2. Effect of $D(s)$ is easily sketched.
3. If minimum phase (all poles and zeros in LHP), then by Bode's gain-phase relationship, only gain plot is needed.

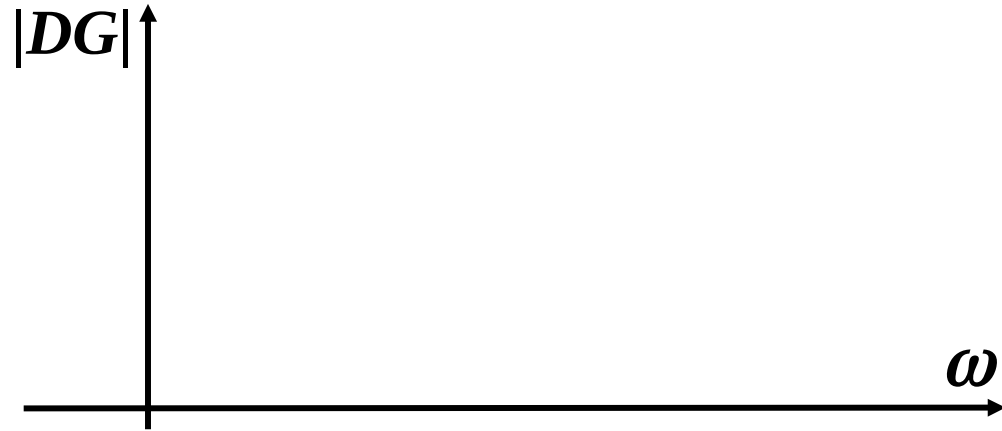
4. Can determine system type.



$$K_p = \lim_{s \rightarrow 0} D(s)G(s)$$

Steady-state error to unit step:

$$e(\infty) = \frac{1}{1 + K_p}$$



$$K_v = \lim_{s \rightarrow 0} sD(s)G(s)$$

$$D(s)G(s) = \frac{K}{s} \frac{(s + z_1)(s + z_2) \cdots (s + z_m)}{(s + p_1)(s + p_2) \cdots (s + p_n)}$$

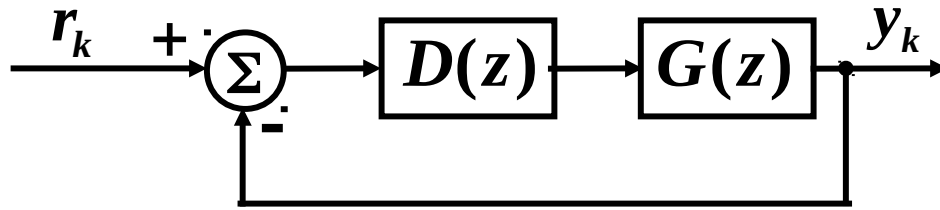
$$\lim_{s \rightarrow 0} D(s)G(s) =$$

$$\approx \lim_{s \rightarrow 0} \frac{K_0}{s}$$

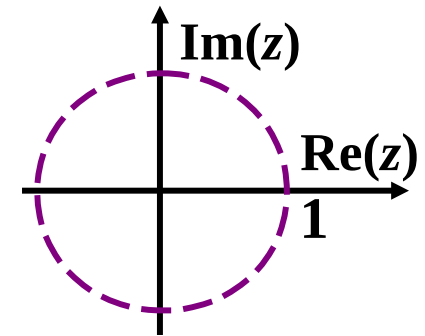
Steady-state error to unit ramp: $e(\infty) = \frac{1}{K_v}$

5. Easy to sketch Nyquist plot and apply stability criterion.

Frequency Response for Discrete Systems



$$G(z) = K \frac{(z + z_{d_1})(z + z_{d_2}) \cdots (z + z_{d_m})}{(z + p_{d_1})(z + p_{d_2}) \cdots (z + p_{d_n})},$$



$$m \leq n$$

$$|G(z)| \Big|_{z=e^{j\omega T}}$$

$$\angle G(z) \Big|_{z=e^{j\omega T}}$$

1. Difficult to sketch by hand.
2. Difficult to see the effect of $D(z)$ on system.

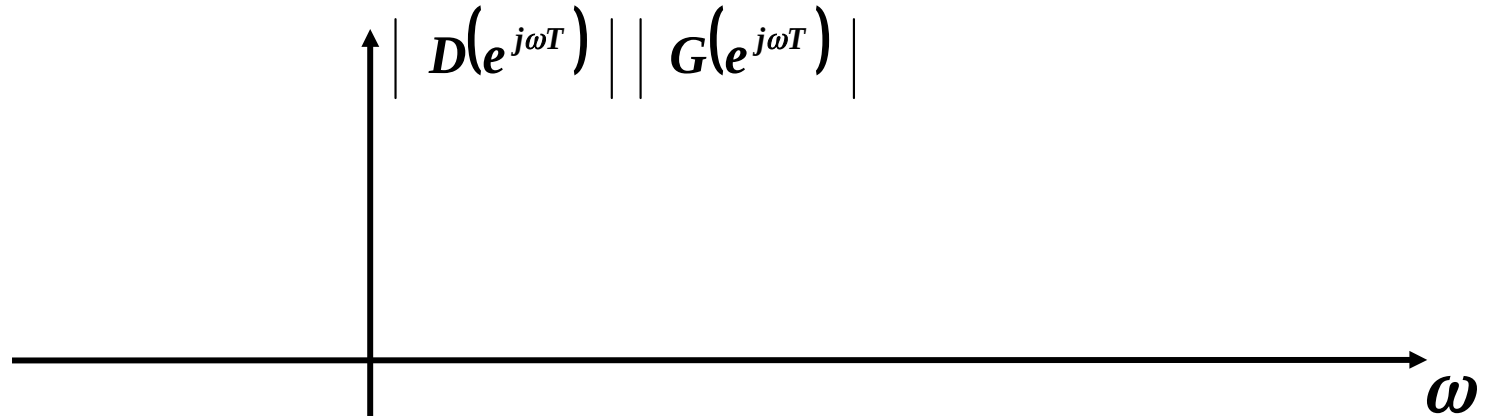
Generally use computers to plot discrete-time frequency response.

- Frequency plots of $G(z)$ are for $z = e^{j\omega T}$ for $0 \leq \omega T \leq \pi$.

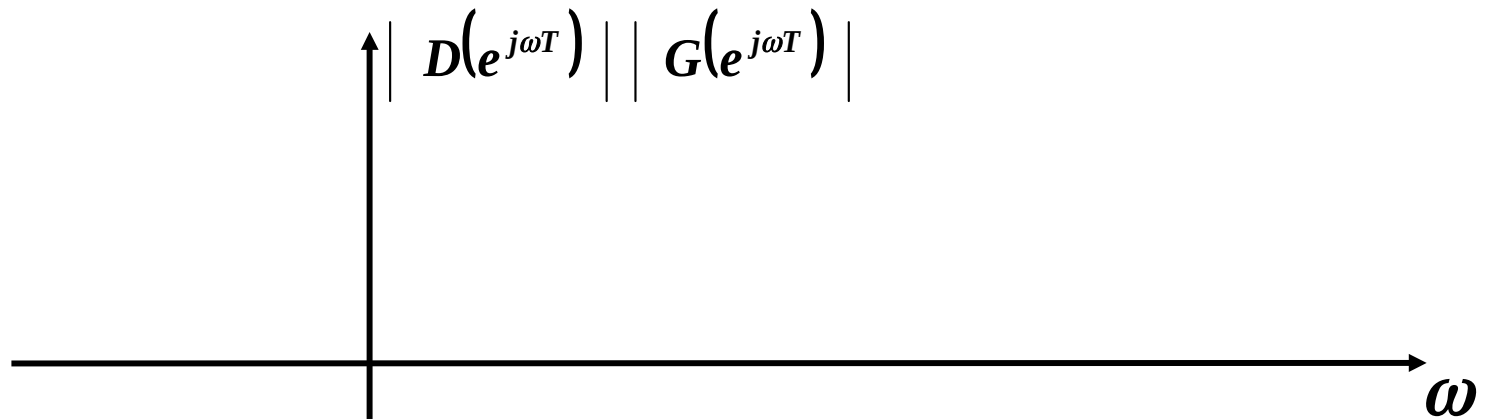


- Discuss Example 7.8 of text.

3. Can still easily determine system type, as well as error constants (K_p or K_v), from frequency response plots.



$$\begin{aligned} K_p &= \lim_{z \rightarrow 1} D(z)G(z) \\ &= \\ &= \lim_{\omega \rightarrow 0} D(e^{j\omega T})G(e^{j\omega T}) \end{aligned}$$



$$K_v = \lim_{z \rightarrow 1} \frac{(z - 1)D(z)G(z)}{Tz}$$

Now, $e^{j\omega T} \approx 1 + j\omega T$ for $\omega T \rightarrow 0$

$$K_v = \lim_{\omega \rightarrow 0} \frac{j\omega T D(\cdot)G(\cdot)}{T(1 + j\omega T)}$$

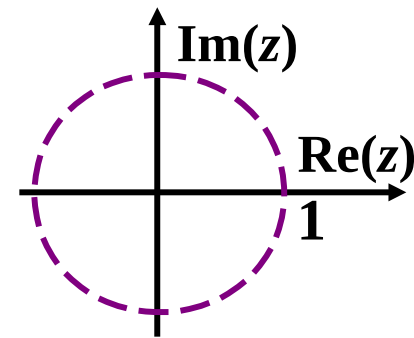
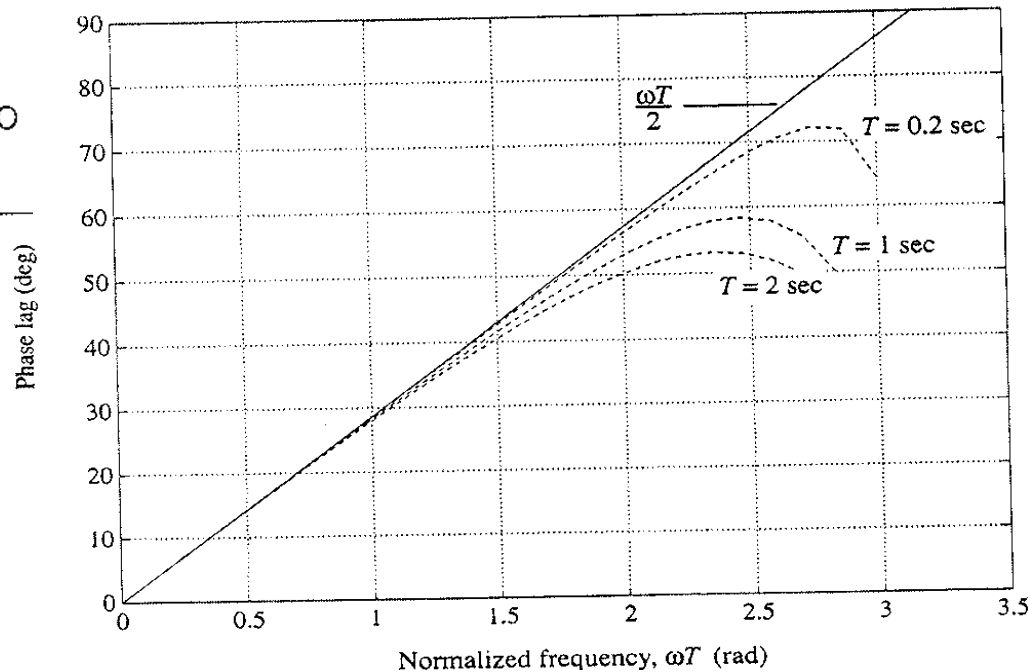
4. Easy to sketch Nyquist plot and apply stability criterion.

Phase Lag Due to Sampling

$$\Delta\phi = \frac{\omega T}{2}$$

good approximation for $\omega T < \frac{\pi}{2} \approx 1.6$

Figure 7.17
Phase lag due to sampling

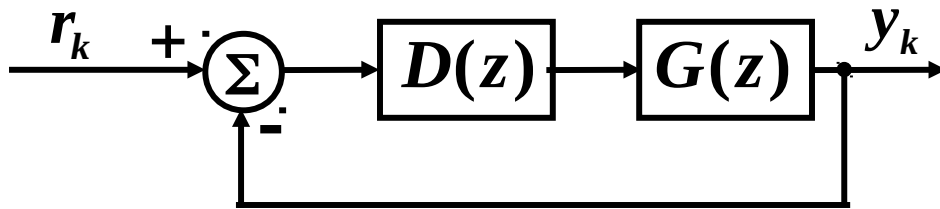


If have plotted $|G(j\omega)|$ and $\angle G(j\omega)$ (easy to do by hand), then find PM of $G(s)$.

$$\text{Then: } PM \text{ of } G(z) \approx PM \text{ of } G(s) - \frac{\omega T}{2}$$

Gain and Phase Margins

Defined exactly the same way for discrete-time systems as for continuous-time systems.



Critical condition: $|DG| = 1$, $\angle DG = 180^\circ$

GM: Factor by which can increase open-loop gain before system becomes unstable.

– find ω_{180} where $\angle DG = 180^\circ$ on the frequency response plot of the phase.

– then read off $|DG|$ at ω_{180} $GM = \frac{1}{|DG|} \Big|_{\omega_{180}}$

Revisit Example 7.8 of text:

$$G(s) = \frac{1}{s(s+1)} \quad \text{infinite } GM$$

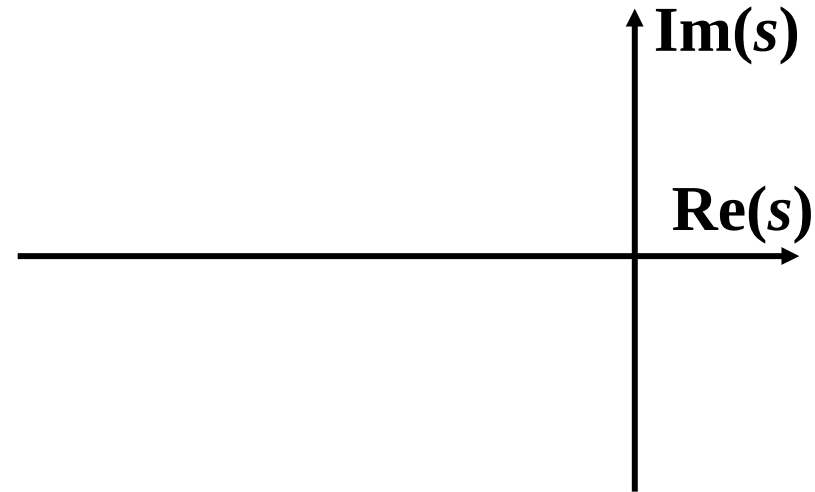
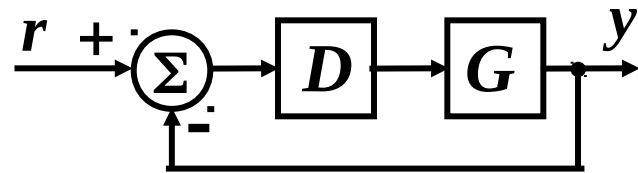
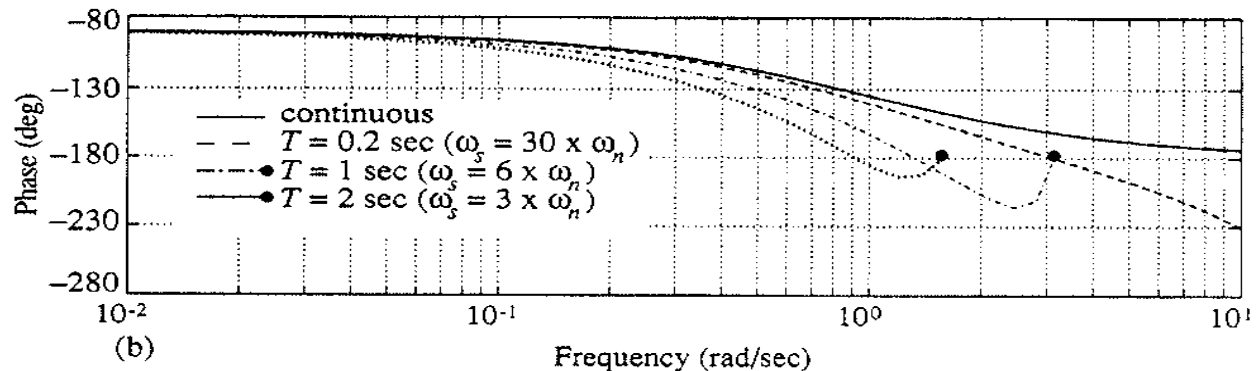
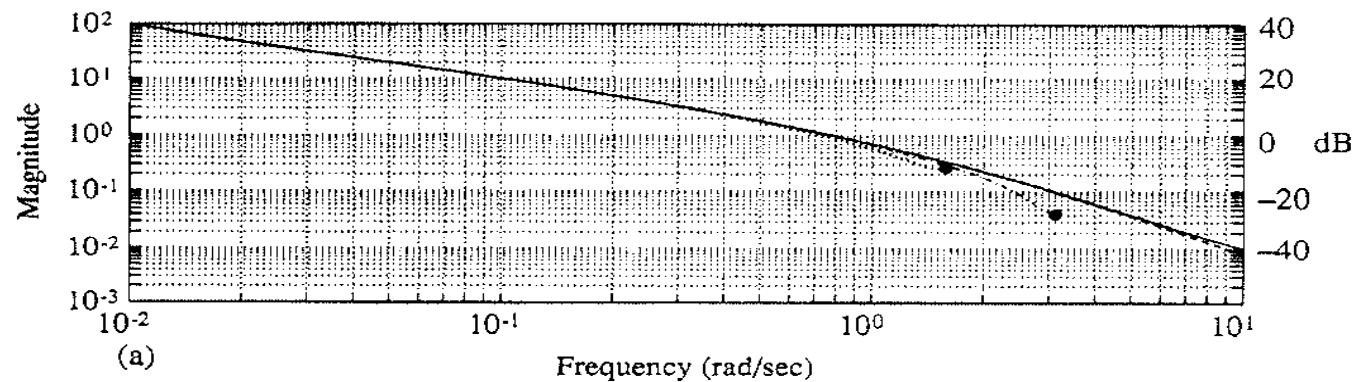


Figure 7.16
of text.

For $G(z)$ with
 $T = 0.2$ sec:

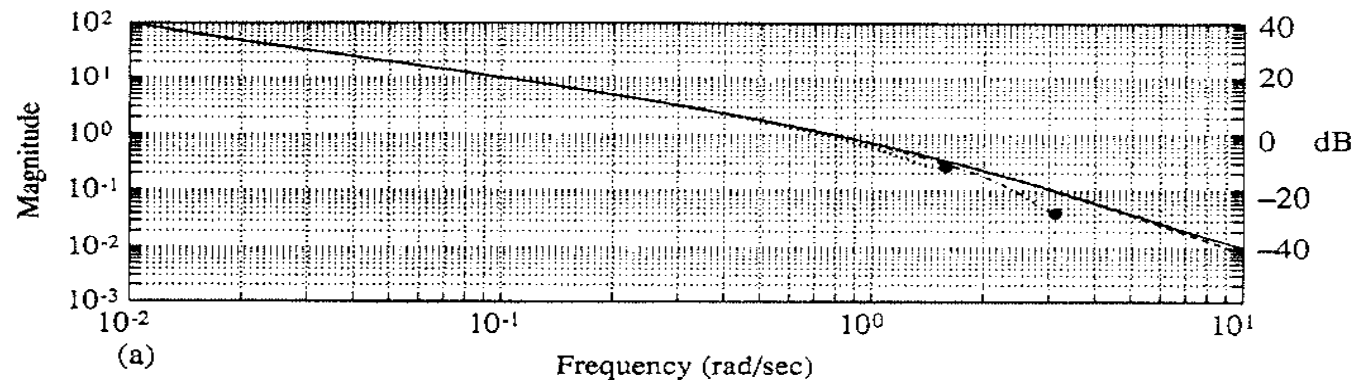
$$\omega_{180} \approx$$



$$T = 1 \text{ sec: } \omega_{180} \approx$$

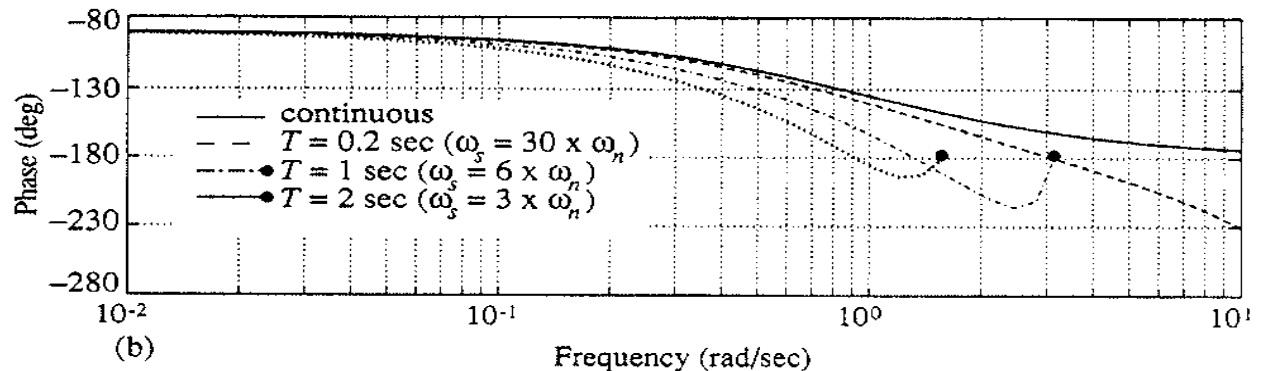
$$T = 2 \text{ sec: } \omega_{180} \approx$$

Figure 7.16
of text.



As $T \uparrow$, $GM \downarrow$.

If stable,
 $GM > 1$.



PM: Find ω_c where $|DG| = 1$.

$$PM = \angle DG \Big|_{\omega_c} + 180^\circ$$

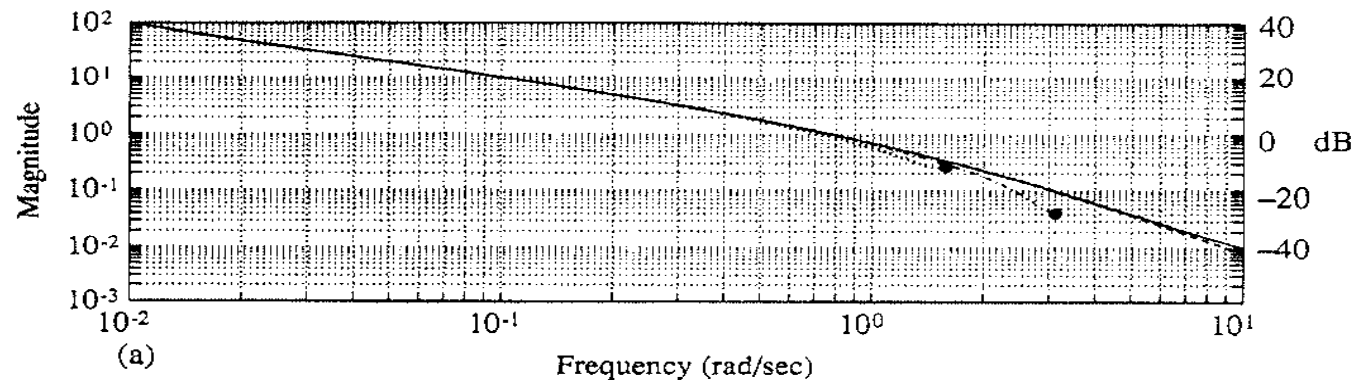
If stable, $PM > 0$.

$$G(s): \angle DG \Big|_{\omega_c} \approx$$

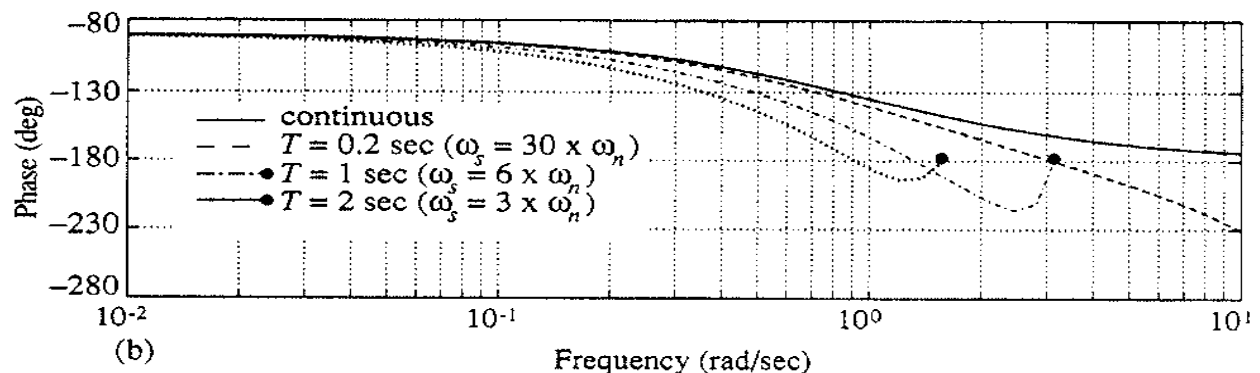
$$PM =$$

Revisit Example 7.8 of text:

Figure 7.16
of text.



$$\omega_c \approx 0.7 \text{ rad/s}$$



$$G(z): \quad T = 0.2 \text{ sec}: \quad \square \quad \left. \frac{DG}{d\omega} \right|_{\omega_c} \approx \quad PM =$$

$$T = 1 \text{ sec}: \quad \square \quad \left. \frac{DG}{d\omega} \right|_{\omega_c} \approx \quad PM =$$

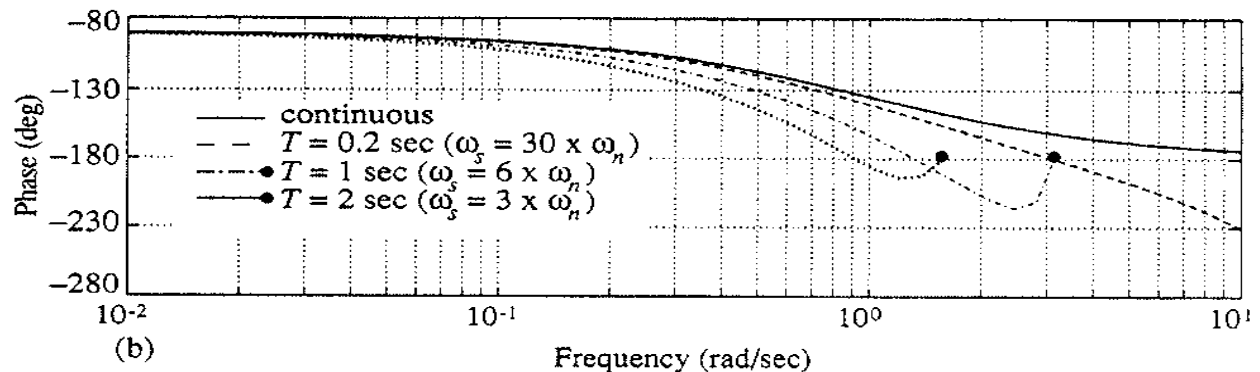
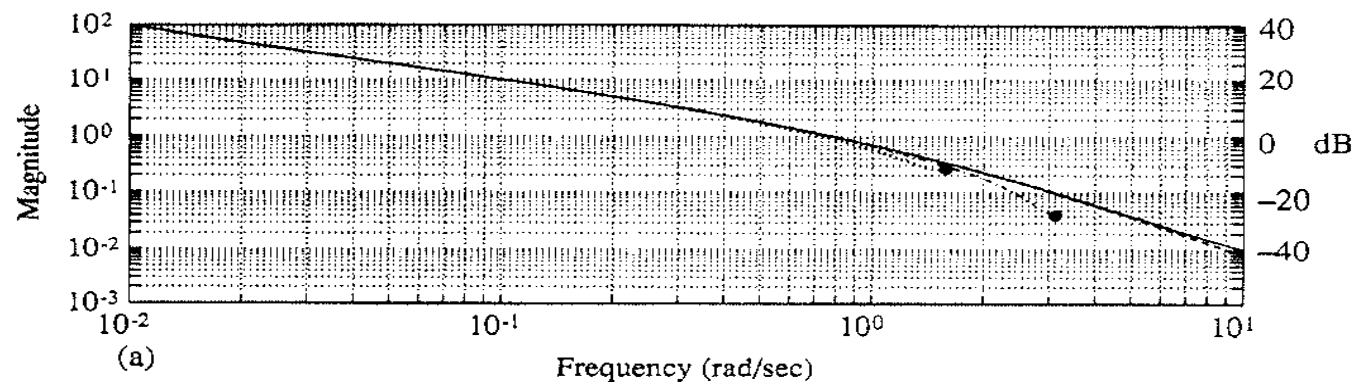
$$T = 2 \text{ sec}: \quad \square \quad \left. \frac{DG}{d\omega} \right|_{\omega_c} \approx \quad PM =$$

Figure 7.16
of text.

$$\omega_c \approx 0.7 \text{ rad/s}$$

As $T \uparrow$, $PM \downarrow$.

If stable,
 $PM > 0$.



- For system to be stable, need $PM > 0$ & $GM > 1$.
- Rule of thumb: $PM > 0$ & $GM > 1 \Rightarrow$ stable.
 - Works most of the time.

