

Introducing a Reference Input

- **Designing M and N**
 - Design so that estimator performance is independent of reference input
 - General design of choosing overall system zeros
- **Output error command structure**
 - Difference between transfer function and state-space compensator design approaches

Introducing a Reference Input

Thus far, we have assumed that the reference input is zero. That is, we want $y = 0$, thus in steady state, $x = 0$ and $u = 0$. If there are disturbances that cause $x \neq 0$ and $y \neq 0$, then we have $u = -Kx$ to reject the disturbances to bring $x \approx 0$, $y \approx 0$.

Now suppose we want $r \neq 0$.

What is the best way to introduce r into the system?

Controller Equations

$$\hat{\mathbf{x}}(k+1) = (\Phi - \Gamma\mathbf{K} - \mathbf{L}_p\mathbf{H}) \hat{\mathbf{x}}(k) + \mathbf{L}_p y(k) + \mathbf{M}r(k)$$

$$u(k) = -\mathbf{K}\hat{\mathbf{x}}(k) + \mathbf{N}r(k)$$

Since r is an external signal, \mathbf{M} and \mathbf{N} do not affect the closed-loop characteristic equation. That is, the poles of the system are fixed, hence \mathbf{M} and \mathbf{N} do not affect stability.

\mathbf{M} and \mathbf{N} affect the locations of zeros, which affect the transient response of the system.

\mathbf{M} and \mathbf{N} also affect steady-state error.

Designing M and N

The most common way is to select M and N such that

$$\tilde{x} \perp r$$

The idea here is that if the estimator is good, it should be good independent of any external excitation.

This is not the most general way of designing M and N though. The most general way is to design n zeros, then solve for M and N to yield those zeros.

Estimator Error Dynamics Equation

$$\tilde{\mathbf{x}}(k+1) = \mathbf{x}(k+1) - \hat{\mathbf{x}}(k+1)$$

$$\tilde{\mathbf{x}}(k+1) = (\Phi - \mathbf{L}_p \mathbf{H})\tilde{\mathbf{x}}(k) + (\Gamma \mathbf{N} - \mathbf{M})r(k)$$

If r does not affect estimator performance, then

$$\mathbf{M} = \Gamma \mathbf{N}$$

Controller Equations

Originally:

$$\hat{\mathbf{x}}(k+1) = (\Phi - \Gamma\mathbf{K} - \mathbf{L}_p\mathbf{H})\hat{\mathbf{x}}(k) + \mathbf{L}_p\mathbf{y}(k) + \mathbf{M}r(k)$$
$$u(k) = -\mathbf{K}\hat{\mathbf{x}}(k) + \mathbf{N}r(k)$$

Now:

$$\hat{\mathbf{x}}(k+1) = (\Phi - \mathbf{L}_p\mathbf{H})\hat{\mathbf{x}}(k) - \Gamma\mathbf{K}\hat{\mathbf{x}}(k) + \mathbf{M}r(k) + \mathbf{L}_p\mathbf{y}(k)$$

$$\hat{\mathbf{x}}(k+1) = (\Phi - \mathbf{L}_p\mathbf{H})\hat{\mathbf{x}}(k) + \Gamma u(k) + \mathbf{L}_p\mathbf{y}(k)$$
$$u(k) = -\mathbf{K}\hat{\mathbf{x}}(k) + \mathbf{N}r(k)$$

Designing N

For a constant r , we want $e_{ss} = 0$ so that $y_{ss} = r_{ss} = r$

Plant:
$$\begin{aligned} \mathbf{x}(k+1) &= \Phi \mathbf{x}(k) + \Gamma u(k) \\ y(k) &= \mathbf{H} \mathbf{x}(k) + J u(k) \end{aligned}$$

In steady-state: $\mathbf{x}(k+1) = \mathbf{x}(k) = \mathbf{x}_{ss}$

$$\mathbf{x}_{ss} = \Phi \mathbf{x}_{ss} + \Gamma u_{ss}$$

$$y_{ss} = \mathbf{H} \mathbf{x}_{ss} + J u_{ss}$$

We want $y_{ss} = r_{ss}$, then:

u_{ss} is some constant scalar:

$$u_{ss} = N_u r_{ss}$$

\mathbf{x}_{ss} is some constant vector:

$$\mathbf{x}_{ss} = \mathbf{N}_x r_{ss}$$

$$\begin{bmatrix} \mathbf{N}_x \\ N_u \end{bmatrix} = \begin{bmatrix} \Phi - \mathbf{I} & \Gamma \\ \mathbf{H} & J \end{bmatrix}^{-1} \begin{bmatrix} \mathbf{0} \\ 1 \end{bmatrix}$$

Now, we can write

$$u = -K\hat{x} + Nr$$

In steady-state,

$$u_{ss} = -Kx_{ss} + Nr$$

Thus,

$$u = -K\hat{x} + Kx_{ss} + u_{ss}$$

$$u = -K\hat{x} + (N_u + KN_x) r$$

Overall System State Equations with Non-Zero Reference Input

Plant $G(z)$:

$$\mathbf{x}(k+1) = \Phi \mathbf{x}(k) + \Gamma u(k)$$

$$y(k) = \mathbf{H} \mathbf{x}(k) + J u(k)$$

Compensator $D(z)$:

$$\hat{\mathbf{x}}(k+1) = (\Phi - \mathbf{L}_p \mathbf{H}) \hat{\mathbf{x}}(k) + \Gamma u(k) + \mathbf{L}_p y(k)$$

$$u(k) = -\mathbf{K} \hat{\mathbf{x}}(k) + N r(k)$$

$$\mathbf{M} = \Gamma N$$

$$\tilde{\mathbf{x}}(k+1) = (\Phi - \mathbf{L}_p \mathbf{H}) \tilde{\mathbf{x}}(k)$$

$$N = N_u + \mathbf{K} N_x$$

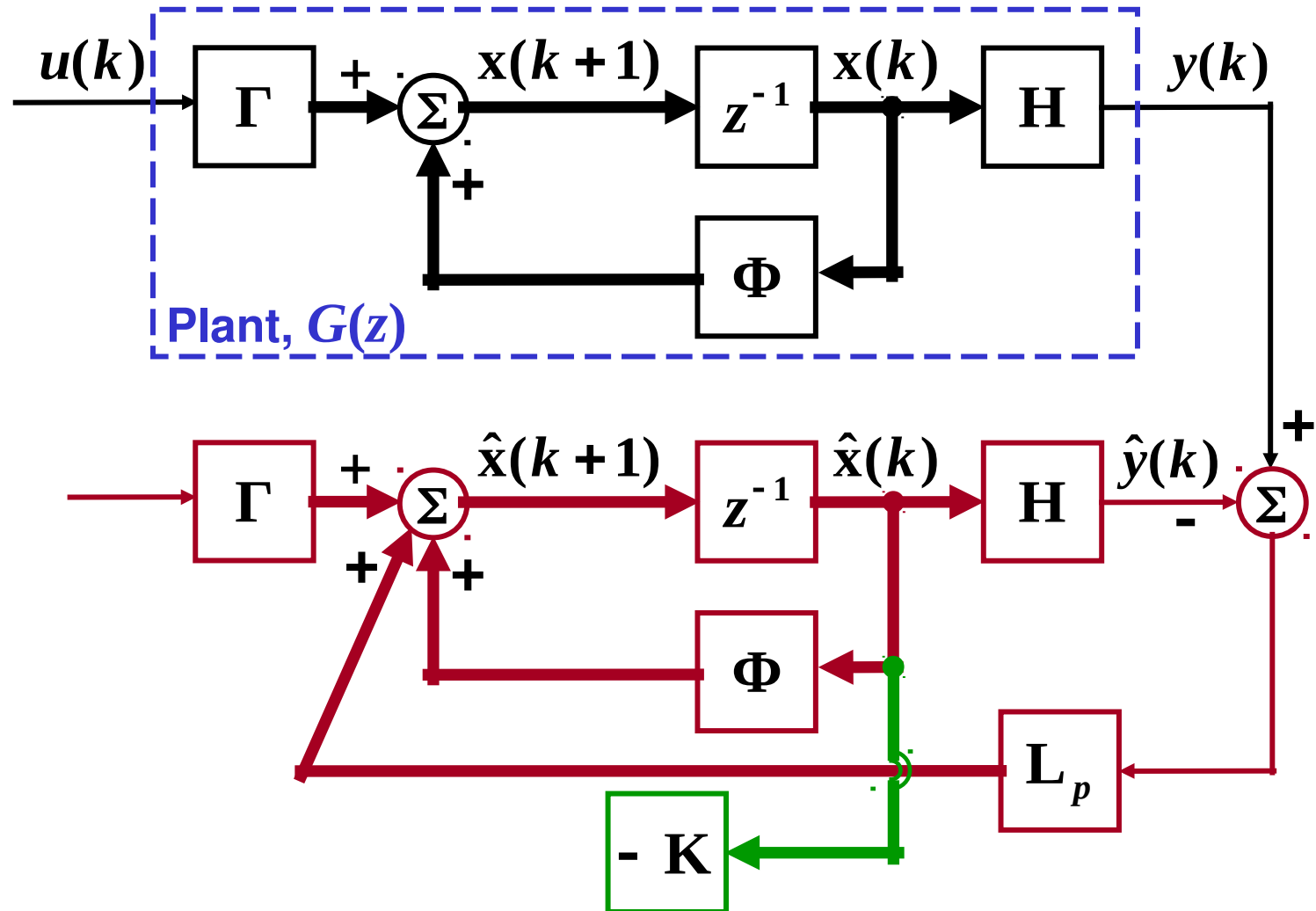
$$\begin{bmatrix} N_x \\ N_u \end{bmatrix} = \begin{bmatrix} \Phi - \mathbf{I} & \Gamma \\ \mathbf{H} & J \end{bmatrix}^{-1} \begin{bmatrix} \mathbf{0} \\ 1 \end{bmatrix}$$

$$\mathbf{K} : \det(z\mathbf{I} - \Phi + \Gamma \mathbf{K}) = \alpha_c(z)$$

$$\mathbf{L}_p : \det(z\mathbf{I} - \Phi + \mathbf{L}_p \mathbf{H}) = \alpha_e(z)$$

State-Command Structure

Drives the plant model in the estimator with the same inputs that are applied to the actual plant.



General Case Where Zeros Are Designed

$$\hat{\mathbf{x}}(k+1) = (\Phi - \Gamma\mathbf{K} - \mathbf{L}_p\mathbf{H}) \hat{\mathbf{x}}(k) + \mathbf{L}_p y(k) + \mathbf{M}r(k)$$

$$u(k) = -\mathbf{K}\hat{\mathbf{x}}(k) + \mathbf{N}r(k)$$

1. Consider the case of only state feedback with no estimator, but with a non-zero reference input:

$$\mathbf{x}(k+1) = \Phi\mathbf{x}(k) + \Gamma u(k)$$

$$y(k) = \mathbf{H}\mathbf{x}(k)$$

What are the zeros?

$$\begin{bmatrix} zI - \Phi + \Gamma K & -\Gamma \\ 0 & NR(z) \end{bmatrix} X(z) = 0$$

$$\det \begin{bmatrix} zI - \Phi & -\Gamma \\ 0 & 0 \end{bmatrix} = 0$$

2. Consider the case of the combined controller and estimator with non-zero reference input:

$$\hat{x}(k+1) = (\Phi - \Gamma K - L_p H) \hat{x}(k) + L_p y(k) + M r(k)$$

$$u(k) = -K \hat{x}(k) + N r(k)$$

What are the zeros from $r(k)$ to $u(k)$?

Without loss of generality, assume $y(k) = 0$.

$$\begin{bmatrix} zI - \Phi + \Gamma K + L_p H \\ -K \end{bmatrix} - \frac{M}{N} \begin{bmatrix} \hat{X}(z) \\ NR(z) \end{bmatrix} = 0$$

$$\det \begin{bmatrix} zI - \Phi + \Gamma K + L_p H - \frac{M}{N} K \\ \hline \end{bmatrix} = \gamma(z) = 0$$

$$\det \begin{bmatrix} zI - A + \frac{M}{N} (-K) \\ \hline \end{bmatrix} = \gamma(z) = 0$$

Compare with estimator problem: $\det(zI - \Phi + LH) = 0$

We can choose zeros of compensator $D(z)$, and hence

$$H(z) = \frac{Y(z)}{R(z)}$$

If $(A, -K)$ is observable, then can choose desired zeros, then form $y(z) = 0$ and solve for M/N .

$$\begin{bmatrix} 1 & -K \\ 1 & -KA \\ 1 & \vdots \\ 1 & -KA^{n-1} \end{bmatrix}$$

$$\frac{Y(z)}{R(z)} = \eta \frac{y(z)b(z)}{\alpha_e(z)\alpha_c(z)}$$

Special case:

$$\mathbf{M} = \Gamma N$$

$$\tilde{\mathbf{x}} \perp r$$

What are the zeros from $r(k)$ to $u(k)$?

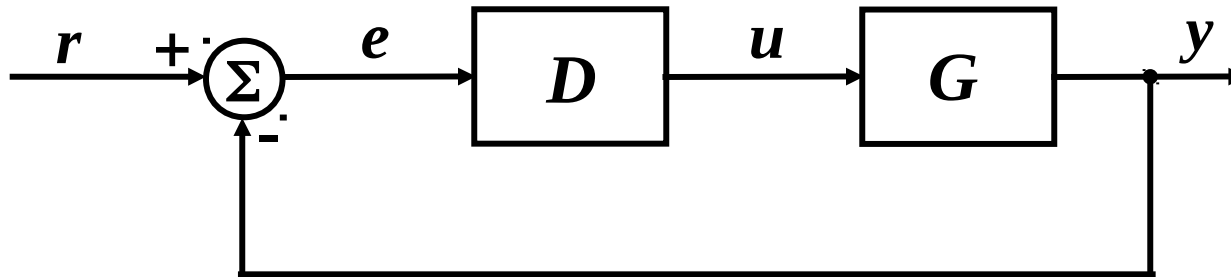
$$\det \begin{bmatrix} z\mathbf{I} - \Phi + \Gamma\mathbf{K} + \mathbf{L}_p\mathbf{H} - \frac{\mathbf{M}}{N}\mathbf{K} \\ \mathbf{0} \end{bmatrix} = \gamma(z) = 0$$

$$\gamma(z) = \det(z\mathbf{I} - \Phi + \mathbf{L}_p\mathbf{H})$$

Overall poles of the system are: $\alpha_e(z)\alpha_c(z) = 0$

$$\frac{Y(z)}{R(z)} = \eta \frac{b(z)}{\alpha_c(z)}$$

Output Error Command Structure



The compensator equations:

$$\hat{\mathbf{x}}(k+1) = (\Phi - \Gamma\mathbf{K} - \mathbf{L}_p\mathbf{H}) \hat{\mathbf{x}}(k) + \mathbf{L}_p y(k) + \mathbf{M}r(k)$$

$$u(k) = -\mathbf{K}\hat{\mathbf{x}}(k) + \mathbf{N}r(k)$$

Since we need all r and y terms only in terms of $(r - y)$, we must have $\mathbf{M} = -\mathbf{L}_p$, $\mathbf{N} = \mathbf{I}$ and the compensator equations become:

$$\hat{\mathbf{x}}(k+1) = (\Phi - \Gamma\mathbf{K} - \mathbf{L}_p\mathbf{H}) \hat{\mathbf{x}}(k) - \mathbf{L}_p(r(k) - y(k))$$

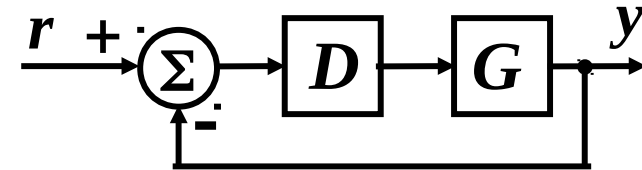
$$u(k) = -\mathbf{K}\hat{\mathbf{x}}(k)$$

What are the zeros of $D(z)$?

$$\begin{bmatrix} z\mathbf{I} - \Phi + \Gamma\mathbf{K} + \mathbf{L}_p\mathbf{H} & \mathbf{L}_p \\ \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{bmatrix} \hat{\mathbf{X}}(z) \\ \mathbf{E}(z) \end{bmatrix} = \mathbf{0}$$

$$\det \begin{bmatrix} zI - \Phi & L_p \\ 0 & K(zI - \Phi)^{-1}L_p \end{bmatrix} = 0$$

$$\det(zI - \Phi)K(zI - \Phi)^{-1}L_p = 0$$



These are zeros of the compensator, and they will be the zeros of the overall system unless they are cancelled by plant poles.

$$H(z) = \frac{Y(z)}{R(z)} = \frac{\gamma(z)b(z)}{\alpha_e(z)\alpha_c(z)}$$

Difference Between Transfer Function and State-Space Compensator Design Approaches

Drawback of transfer function approach:

Zeros of the compensator $D(z)$ become zeros of the overall system $H(z)$.

Advantage of state-space compensator with $\mathbf{M} = \mathbf{\Gamma}N$:

Overall transfer function of the system only has plant zeros. The compensator does not add any additional zeros. This makes it often easier to predict the response.