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## **Combined Control Law & Estimator**

- Separation principle
- What is D(z)?

Example

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## **Combined Control Law & Estimator**

When all states are measured, the control law design is:

$$x(k+1) = \Phi x(k) + \Gamma u(k)$$
  
 $y(k) = Hx(k) + Ju(k)$ 
 $\Rightarrow u(k) = -Kx(k)$ 

But if not all states are measured, or measurements are noisy, design a state estimator with desired estimator error dynamics:

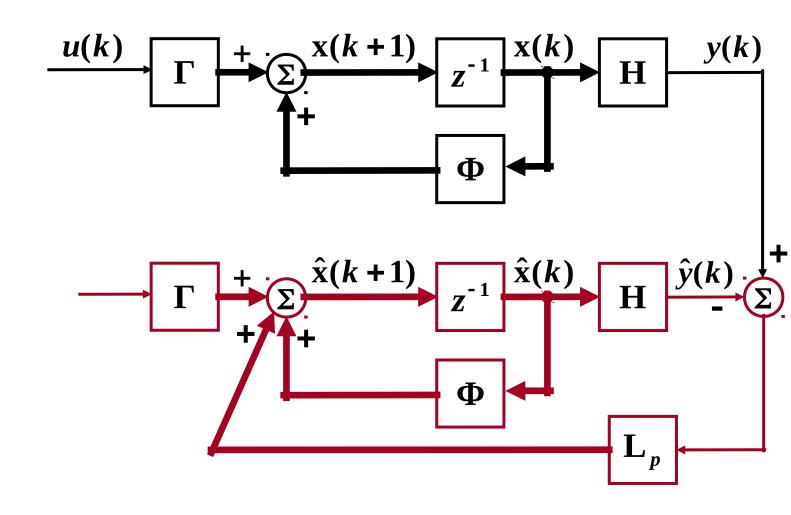
 $\widetilde{\mathbf{x}}(k+1) = (\Phi - \mathbf{L}_p \mathbf{H})\widetilde{\mathbf{x}}(k)$ 

and then use:  $u(k) = -K\hat{x}(k)$ 

- But state feedback was designed assuming  $\, x \,$  is fedback, not  $\, \hat{x} . \,$
- Is it really okay to feedback \$\hat{x}\$
- Or do we need to completely redesign  ${\bf K}$ , taking into account now that we are feeding back  ${\bf x}$  and not  $\hat{{\bf x}}$

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## **Block Diagram of Combined Control & Estimator System**



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# Analysis of the Combined Control & Estimator System with $\hat{\mathbf{x}}$ Fedback

$$x(k+1) = \Phi x(k) + \Gamma u(k) = \Phi x(k) - \Gamma K \hat{x}(k)$$

$$x(k+1) = \Phi x(k) - \Gamma K(x(k) - \tilde{x}(k))$$

along with estimator error dynamics equation:

$$\widetilde{\mathbf{x}}(k+1) = (\Phi - \mathbf{L}_p \mathbf{H})\widetilde{\mathbf{x}}(k)$$

yields the overall system dynamics (control and estimator):

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## **Overall Closed-Loop Characteristic Equation**

$$\det(z\mathbf{I}-F)=0$$

$$\det_{\begin{bmatrix} \\ \\ \end{bmatrix}}^{\begin{bmatrix} \\ \\ \end{bmatrix}} \underbrace{0} \quad zI - \Phi + L_pH_{\begin{bmatrix} \\ \\ \end{bmatrix}}^{\begin{bmatrix} \\ \\ \end{bmatrix}} = 0$$

$$\det(zI - \Phi + \Gamma K) \det(zI - \Phi + L_pH) = 0$$

Poles of the overall closed-loop system:

desired control poles + desired estimator poles

#### **Separation Principle**

Controller design and estimator design can be carried out independently.

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# What is D(z)?

$$\hat{\mathbf{x}}(k+1) = \Phi \hat{\mathbf{x}}(k) + \Gamma u(k) + \mathbf{L}_{p}(y(k) - \hat{y}(k))$$

$$\hat{\mathbf{x}}(k+1) = (\Phi - \Gamma \mathbf{K} - \mathbf{L}_p \mathbf{H}) \hat{\mathbf{x}}(k) + \mathbf{L}_p \mathbf{y}(k)$$

$$u(k) = -\mathbf{K} \hat{\mathbf{x}}(k)$$

## Compare with standard state-space equation:

$$x(k+1) = \Phi x(k) + \Gamma u(k)$$
$$y(k) = Hx(k) + Ju(k)$$

$$G(z) = \frac{b(z)}{a(z)} = H(zI - \Phi)^{-1}\Gamma + J$$

## Compensator transfer function is then:

$$D(z) = \frac{U(z)}{Y(z)} = -K(zI - \Phi + \Gamma K + L_pH)^{-1}L_p$$

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#### **Comments**

- Poles and zeros of D(z) are never specified in the state-space compensator (controller & estimator) design.
- The compensator D(z) itself may in fact be unstable!
  - Overall closed-loop system of plant and compensator is stable for stable controller and estimator designs, but the compensator itself may have unstable poles.
- The transfer function D(z) from the state-space controller/estimator design has similarities with compensators designed using Root Locus or Frequency Response/Bode methods.

## **Example**

$$G(s) = \frac{1}{s(s+1)} = \frac{Y(s)}{U(s)} \quad \xrightarrow{T = 2\sec} \quad G(z) = 1.135 \frac{z + 0.523}{(z-1)(z-0.135)}$$

$$\Phi = \begin{bmatrix} 1 & 1 - e^{-T} \\ 0 & e^{-T} \end{bmatrix} = \begin{bmatrix} 1 & 0.865 \\ 0 & 0.135 \end{bmatrix},$$

$$\Gamma = \begin{bmatrix} T + e^{-T} - 1 \\ 1 - e^{-T} \end{bmatrix} = \begin{bmatrix} 1.135 \\ 0.865 \end{bmatrix},$$

$$\mathbf{H} = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad J = \mathbf{0}$$

State feedback: 
$$\zeta = 0.3$$
,  $\omega_n = 1$  rad/sec  $\Rightarrow K = \begin{bmatrix} 0.9623 & 0.4693 \end{bmatrix}$ 

Estimator: 
$$\zeta = 0.5$$
,  $\omega_n = 1.5 \text{ rad/sec} \implies L_p = \begin{bmatrix} 1.5173 \\ 0.1385 \end{bmatrix}$ 

What is the compensator transfer function D(z)?

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Re(z)

Im(z)

$$D(z) = \frac{-1.53z + 0.147}{z^2 + 1.88z + 0.76} = \frac{-1.53(z - 0.096)}{(z + 1.296)(z + 0.585)}$$



