

# Introduction to State-Space Methods

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- **Advantages**
- **Relationship between transfer functions and state equations (review)**
- **Control law design via state feedback**

# State-Space Methods

## Continuous-Time

$$Y(s) = \mathbf{G}(s)U(s)$$

$$\begin{aligned}\dot{\mathbf{x}} &= \mathbf{F}\mathbf{x} + \mathbf{G}u \\ y &= \mathbf{H}\mathbf{x} + Ju\end{aligned}$$

## Discrete-Time

$$Y(z) = G(z)U(z)$$

$$\begin{aligned}\mathbf{x}(k+1) &= \Phi\mathbf{x}(k) + \Gamma u(k) \\ y(k) &= \mathbf{H}\mathbf{x}(k) + Ju(k)\end{aligned}$$

$$\begin{aligned}\Phi &= e^{\mathbf{F}T} \\ \Gamma &= \int_0^T e^{\mathbf{F}\eta} d\eta \mathbf{G}\end{aligned}$$

## Advantages of state-space methods

1. Easier to solve numerically.
2. Transfer functions are only for LTI systems.
3. State-space methods are easily extendible to MIMO.

# Relationship Between Transfer Functions and State Equations

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$$G(z) = \frac{Y(z)}{U(z)} = H(zI - \Phi)^{-1} \Gamma + J$$

poles:  $\det(zI - \Phi) = 0$

zeros:  $\det \begin{bmatrix} zI - \Phi & -\Gamma \\ H & J \end{bmatrix} = 0$

$$G(z) = \frac{\det \begin{bmatrix} zI - \Phi & -\Gamma \\ H & J \end{bmatrix}}{\det(zI - \Phi)}$$

# Control Law Design via State Feedback

**For now, assume all states are measured and  $r = 0$ .**

**Suppose uncompensated system:**

$$\mathbf{x}(k+1) = \Phi \mathbf{x}(k) + \Gamma u(k)$$

$$y(k) = H \mathbf{x}(k) + J u(k)$$

**does not perform satisfactorily.**

**Then:  $u(k) = -K \mathbf{x}(k)$**

$$= - \begin{bmatrix} k_1 & k_2 & \cdots & k_n \end{bmatrix} \begin{bmatrix} x_1(k) \\ x_2(k) \\ \vdots \\ x_n(k) \end{bmatrix}$$

$$\begin{aligned}\mathbf{x}(k+1) &= \Phi \mathbf{x}(k) + \Gamma u(k) \\ y(k) &= \mathbf{H} \mathbf{x}(k) + J u(k)\end{aligned}$$

$$u(k) = -\mathbf{K} \mathbf{x}(k)$$

$$\boxed{\mathbf{x}(k+1) = (\Phi - \Gamma \mathbf{K}) \mathbf{x}(k)}$$

Characteristic equation:  $\det(z\mathbf{I} - \Phi + \Gamma \mathbf{K}) = 0 \implies$  compensated poles

Determine desired poles  $p_1, \dots, p_n$

Desired closed-loop characteristic equation:

- Want to set  $n$  poles
- Have  $k_1, \dots, k_n$  gains to design
- $n$  degrees of freedom

**Question:** Is it always possible to choose  $k_1, \dots, k_n$  such that the closed-loop (compensated) poles are at any arbitrary locations?

# Controllability

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- **State feedback control can arbitrarily move poles to any desired locations if and only if the system is controllable.**
- **If the system is not controllable, state feedback control may be able to move some of the poles, but can not move all the poles.**

**Definition: A system is controllable if and only if the controllability matrix is nonsingular, where the controllability matrix is:**

$$\mathbf{C} = \begin{bmatrix} \mathbf{\Gamma} & \mathbf{\Phi\Gamma} & \dots & \mathbf{\Phi^{n-1}\Gamma} \end{bmatrix}$$

$$\det \mathbf{C} \neq 0$$

## Example

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$$G(s) = \frac{1}{s(s+1)} = \frac{Y(s)}{U(s)}$$

1. Find a state-space representation of the system.
  2. Find the discrete state-space representation of the sampled system.
  3. For  $T = 2$  sec, design a state feedback control law such that  $t_r < 2\text{sec}$ ,  $M_p < 40\%$ .
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1. 
$$\frac{Y(s)}{U(s)} = \frac{1}{s^2 + s}$$

$$\dot{x}_1 = y$$

$$\dot{x}_2 = x_1$$

$$\dot{y} = -x_2 + u$$

$$\dot{x}_2 = -x_2 + u$$



## 2. Find the discrete state-space representation of the sampled system.

$$\mathbf{x}(k+1) = \Phi \mathbf{x}(k) + \Gamma u(k)$$

$$y(k) = H \mathbf{x}(k) + J u(k)$$

$$\Phi = e^{\mathbf{F}T}$$

$$\Gamma = \int_0^T e^{\mathbf{F}\eta} d\eta \mathbf{G}$$

$$\mathbf{F} = \begin{bmatrix} 0 & 1 \\ 0 & -1 \end{bmatrix}, \quad \mathbf{G} = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$\Phi = \begin{bmatrix} 1 & 1 - e^{-T} \\ 0 & e^{-T} \end{bmatrix}$$

$$\Phi = e^{\mathbf{F}T} = \begin{bmatrix} \mathbf{1} & \mathbf{1} - e^{-T} \\ \mathbf{0} & e^{-T} \end{bmatrix}, \quad \mathbf{G} = \begin{bmatrix} \mathbf{0} \\ \mathbf{1} \end{bmatrix}$$

$$\Gamma = \int_0^T e^{\mathbf{F}\eta} d\eta \mathbf{G}$$

$$\Gamma = \begin{bmatrix} T + e^{-T} - \mathbf{1} \\ \mathbf{1} - e^{-T} \end{bmatrix}$$

3. For  $T = 2$  sec, design a state feedback control law such that  $t_r < 2\text{sec}$ ,  $M_p < 40\%$ .

For  $T = 2$  sec:  $\Phi = \begin{bmatrix} 1 & 0.865 \\ 0 & 0.135 \end{bmatrix}$ ,  $\Gamma = \begin{bmatrix} 1.135 \\ 0.865 \end{bmatrix}$

Where are uncompensated (open-loop) poles?

Where are desired compensated (closed-loop) poles?

## Desired compensated characteristic polynomial:

What is the actual compensated characteristic polynomial as a function of the state feedback gains?

$$= \det \begin{bmatrix} z - 1 & -0.865 \\ 0 & z - 0.135 \end{bmatrix} + \begin{bmatrix} 1.135k_1 & 1.135k_2 \\ 0.865k_1 & 0.865k_2 \end{bmatrix}$$

$$= \det \begin{bmatrix} z - 1 + 1.135k_1 & -0.865 + 1.135k_2 \\ 0.865k_1 & z - 0.135 + 0.865k_2 \end{bmatrix}$$

$$\det(zI - \Phi + \Gamma K) = z^2 + (1.135k_1 + 0.865k_2 - 1.135)z \\ + (0.595k_1 - 0.865k_2 + 0.135)$$

**Coefficients of  $z$  should always be a linear function of the  $k_i$ 's.**

**Match coefficients with desired compensated characteristic polynomial:**

$$\alpha_c(z) = z^2 + 0.363z + 0.301$$

$$\det(zI - \Phi + \Gamma K) = z^2 + (1.135k_1 + 0.865k_2 - 1.135)z \\ + (0.595k_1 - 0.865k_2 + 0.135)$$

$$K = \begin{bmatrix} 0.962 & 0.469 \end{bmatrix}$$

# Summary of Matching Coefficients Method

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1. Determine and compute the desired compensated characteristic polynomial:

$$\alpha_c(z) = z^n + \alpha_1 z^{n-1} + \dots + \alpha_n.$$

2. Compute the actual compensated characteristic polynomial as a function of the state feedback gains:

$$\det(zI - \Phi + \Gamma K).$$

3. Match coefficients and solve for the  $k_i$ 's.

This is a way of designing the state feedback control gains. May not always work though. If the system is uncontrollable, then it may be impossible to solve for  $k_i$ 's to achieve the desired coefficients of  $\alpha_c(z)$ .