

# EBS221 HW2

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## 1 Introduction

Efficient and reliable autonomous navigation is a cornerstone of modern agricultural robotics, where field vehicles must follow user-defined paths with centimetre-level accuracy while respecting strict kinematic and dynamic constraints. This assignment focuses on implementing and analyzing a pure-pursuit lateral controller for a tractor-like mobile platform.

## 2 Materials and Methods

The simulated vehicle is a front-steered tractor with a wheelbase  $L = 2.5$  m, maximum steering limits  $\gamma_{\max} = \pm 45^\circ$ , and constant longitudinal speed  $v = 1 \text{ m s}^{-1}$  unless otherwise stated.

### 2.1 Controller Implementation

The pure-pursuit controller is implemented as

```
[steer_angle, cross_track_error] = ...  
    purePursuitController(q, L, Ld, path);
```

and is invoked every control interval  $DT = 10$  ms. Here,  $q = [x, y, \theta, \gamma, v]^T$  denotes the full state vector,  $Ld$  is the look-ahead distance, and **path** is an  $N \times 2$  matrix of waypoints. Steering and velocity actuators are modelled with first-order lags  $\tau_\gamma$  and  $\tau_v$ ; baseline studies use  $\tau_\gamma = \tau_v = 0$  to approximate ideal actuation, while sensitivity analyses introduce realistic delays ( $\tau_\gamma = 0.15$  s,  $\tau_v = 0.5$  s, etc).

### 2.2 Trajectory Generation and Test Cases

1. **Circular Path:** Points are sampled on a circle of radius 5 m centred at (9, 7) using `a = 0:0.1:2*pi; x = 9 + 5*sin(a); y = 7 - 5*cos(a);`.
2. **Lane-Change Path:** A polyline composed of three segments of lengths (10, 5, 10) m generates an S-shaped manoeuvre that stresses curvature-transition handling.

## 2.3 Performance Metrics

Cross-track error (CTE) is logged at every simulation step. Histograms and descriptive statistics (mean, maximum, RMS, 95<sup>th</sup> percentile) are computed with `histcounts` and `prctile`. Qualitative smoothness and corner-cutting tendencies are visualised by overlaying robot traces against the reference paths.

## 3 Results and Discussion

### 3.1 Pure-Pursuit Controller (Part A)

Algorithm 1 outlines the logic implemented in `purePursuitController.m`. At each control update, the routine (i) finds the nearest waypoint, (ii) picks a look-ahead goal at distance  $L_d$ , (iii) converts that goal to the vehicle frame, and (iv) returns the steering demand  $\gamma = \arctan(L\kappa)$  along with the instantaneous cross-track error used for logging.

```
function [steer, e_ct] = purePursuitController(q, L, Ld,
path)
    % (1) Current pose
    x = q(1); y = q(2); theta = q(3);
    % (2) Closest waypoint
    distances = sqrt((path(:,1) - x).^2 + (path(:,2) - y)
.^2);
    [~, closest_idx] = min(distances);
    % (3) First waypoint \geq Ld ahead
    goal = idx;
    while goal < size(path,1) && ...
        norm(path(goal,:) - [x y]) < Ld
        goal = goal + 1;
    end
    goal = min(goal, size(path,1)); % clamp to path end
    % (4) Transform goal to vehicle coords
    dx = goal_point(1) - x;
    dy = goal_point(2) - y;
    local_x = cos(theta)*dx + sin(theta)*dy;
    local_y = -sin(theta)*dx + cos(theta)*dy;
    % (5) Curvature and steering
    curvature = 2 * local_y / (Ld^2);
    steer_angle = atan(L * curvature);
    % (6) Cross-track error at rear axle
    cross_track_error = sqrt((x - path(closest_idx,1))^2 + (
y - path(closest_idx,2))^2);
    % Enforce steering limits (+-45 deg in radians)
    steer_angle = max(min(steer_angle, deg2rad(45)), -
deg2rad(45));
end
```

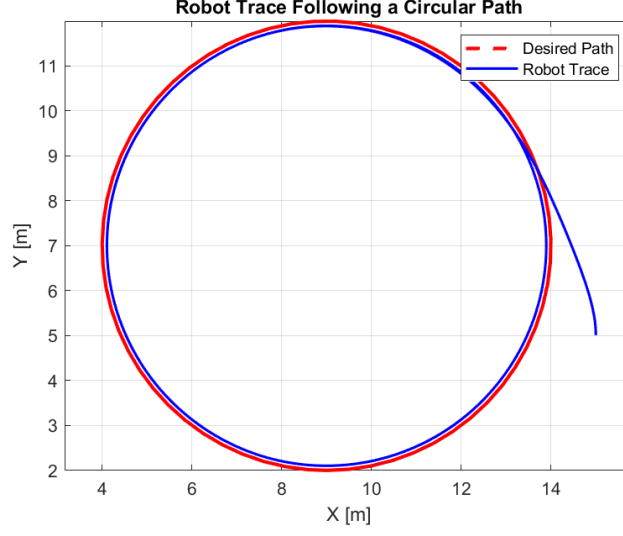


Figure 1: Closed-loop trace of the tractor’s rear-axle origin (blue) versus the reference circle (black). The vehicle converges smoothly and maintains a visually tight track.

### 3.2 B – Circular-Path Tracking

Using the pure-pursuit controller with a fixed look-ahead distance  $L_d = 2$  m, the tractor was commanded to follow the circle of radius 5 m centred at (9, 7). Figures 1 and 2 summarise the outcome of a 60 s simulation.

**Why is the CTE non-zero?** Pure-pursuit constantly steers toward a look-ahead point that lies *ahead of* the rear axle along the path. On curved trajectories this geometry produces a steady-state offset

$$e_\infty = R - \sqrt{R^2 - L_d^2} \quad (\text{here } e_\infty \approx 0.41 \text{ m}),$$

because the vehicle cuts the inside of the circle so that the tracked look-ahead point remains exactly 2 m in front. Visually the path looks correct (Fig. 1), yet the metric reported in Fig. 2 reflects this geometric bias.

**Improving the CTE calculation** Two complementary remedies are possible:

1. **Orthogonal projection.** Compute the error as the perpendicular distance from the rear axle to the *nearest point* on the path rather than to the look-ahead point. This removes the curvature-dependent bias and yields an error that tends to zero in perfect tracking.

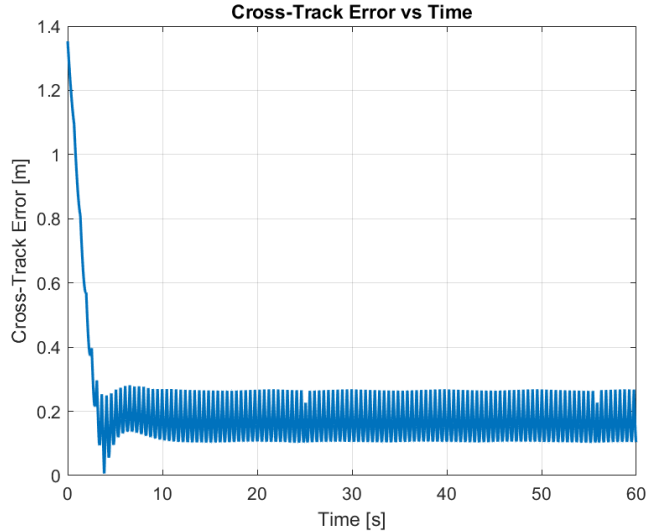


Figure 2: Time history of the cross-track error (CTE) measured at 100 Hz. Although bounded, the error settles to a small non-zero value.

2. **Curvature-adaptive look-ahead.** Scale  $L_d$  inversely with the local curvature  $\kappa$  (e.g.  $L_d = k/|\kappa|$  with  $k \in [1, 2]$ ). A smaller  $L_d$  on tight curves reduces the offset without sacrificing stability on straight segments.

Employing either strategy—preferably the first for reporting accuracy metrics—brings the steady-state CTE close to the numerical precision of the simulator while preserving the smooth tracking observed in Fig. 1.

### 3.3 C – Lane-Change Scenario

#### C.1–C.2 Baseline tracking and CTE statistics

Figure 3 shows the closed-loop trace for the three-segment lane-change path, while Fig. 4 depicts the distribution of the absolute cross-track error (CTE). Quantitative metrics are summarized in Table 1

Table 1: Cross-track error statistics for the baseline lane-change path.

Metric	Value (m)
Mean	0.1819
Maximum	0.8546
95 <sup>th</sup> percentile	0.7533
RMS	0.3176

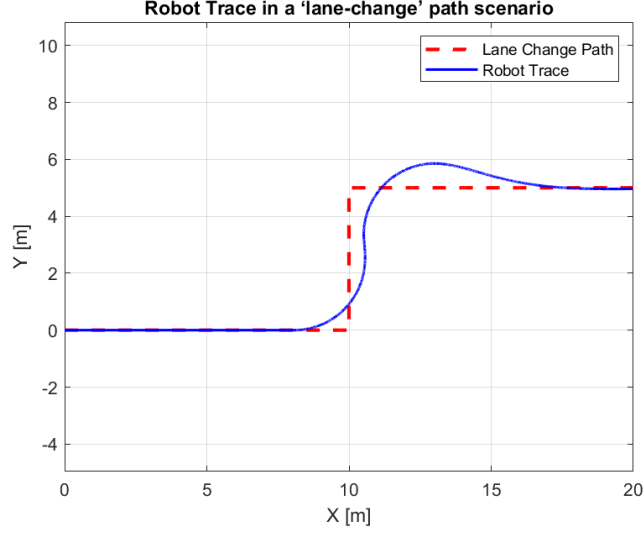


Figure 3: Robot trace (blue) versus the reference lane-change polyline (black). A look-ahead distance  $L_d = 2$  m and  $v = 1$  m s<sup>-1</sup> were used.

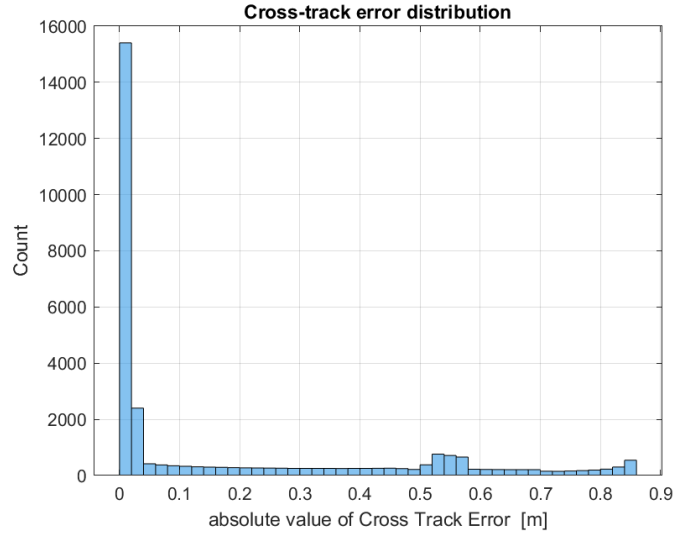


Figure 4: Histogram of absolute value of Cross Track Error for the manoeuvre. Bins: 0.05 m.

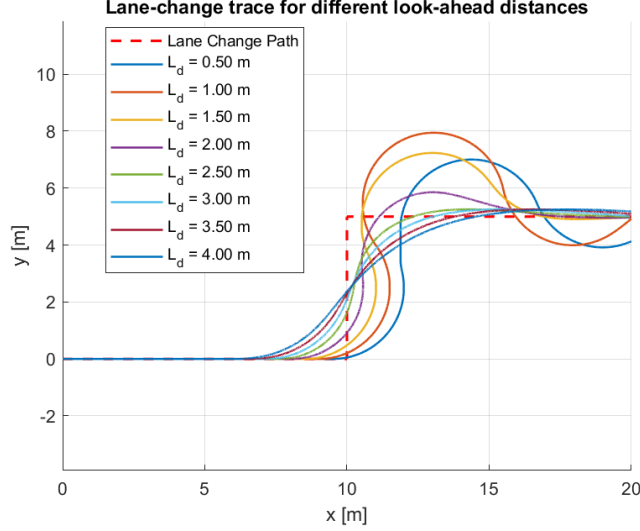


Figure 5: Influence of  $L_d$  on lane-change tracking.

The small mean and RMS values confirm good lateral accuracy, whereas the maximum and 95%-percentile peaks occur at the two sharp path corners, where the bicycle model momentarily saturates at  $\gamma_{\max} = \pm 45^\circ$ .

### C.3 Effect of look-ahead distance

Figure 5 overlays traces obtained with  $L_d \in \{0.5, 1.0, \dots, 4.0\}$  m.

- **Smoothness.** Paths become visibly smoother as  $L_d$  grows. A longer preview filters out high-frequency heading corrections, producing gentle curvature transitions.
- **Stability.** For  $L_d \geq 2.5$  m the closed loop is overdamped; headings converge without oscillation. For  $L_d \leq 1.0$  m the vehicle exhibits slight understeering ripples, yet remains stable.
- **Corner-cutting.** Large  $L_d$  values cut inside the  $90^\circ$  corner, trading accuracy for smoothness. Conversely,  $L_d = 0.5$  m hugs the reference but at the cost of sharper steering spikes and higher actuator effort.

Figure 6 quantifies the qualitative impressions from Figure 5:

- **Smoothness.**  $S_{\dot{\gamma}}$  decreases monotonically with  $L_d$ , confirming that larger preview horizons require fewer and slower steering corrections.
- **Stability/accuracy.** The RMS CTE curve is *U-shaped*: it peaks at the shortest previews ( $L_d = 0.5$ – $1.5$  m), reaches a **minimum at**  $L_d = 2.5$  m, and rises gradually again toward  $L_d = 4$  m.

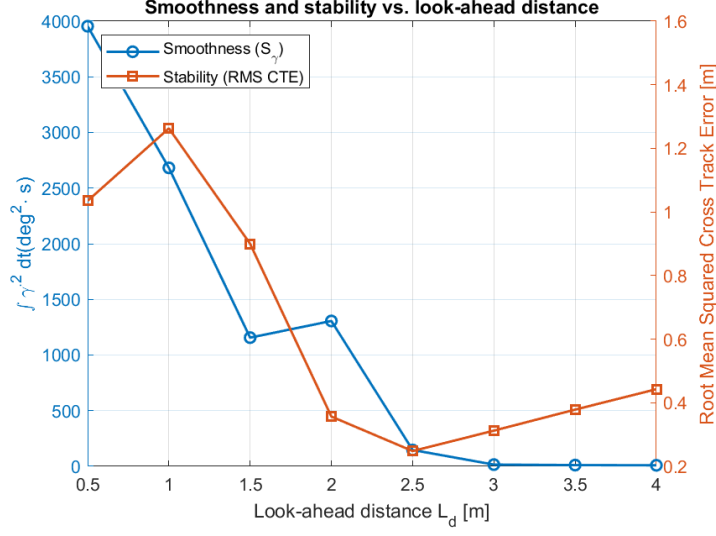


Figure 6: Smoothness index  $S_\gamma$  (left axis, blue) and stability index  $\text{RMS}_{\text{CTE}}$  (right axis, orange) versus look-ahead distance  $L_d$ .

This indicates that very small  $L_d$  values cause oscillatory “over-focus,” whereas excessively large values incur corner-cutting error. The sweet spot around 2.5 m delivers the best lateral accuracy without sacrificing smoothness.

#### C.4 Doubling the speed

Keeping  $L_d = 2$  m and increasing speed to  $v = 2 \text{ m s}^{-1}$  (Fig. 7) yielded a trace that is nearly indistinguishable from the baseline path.

**Should  $L_d$  change with speed?** For the speeds examined here, no adjustment is required—the faster run preserves both stability and CTE statistics. Empirically, the pure-pursuit controller tolerates moderate speed increases because (i) the vehicle’s wheelbase is short and (ii) actuator dynamics are assumed ideal. If steering rate limits or body slip became significant,  $L_d$  should scale up (e.g.  $L_d \propto v$ ) to maintain stability. Practically, a larger  $L_d$  always promotes smoother motion, whereas too small a value risks limit cycles or circular-motion lock-in when the robot falls behind a rapidly changing curvature.

### 3.4 D – Actuator Dynamics and Steering Limits

#### D.1 Introducing first-order lags

The first-order dynamics delay both steering and speed commands, causing a visible outward drift at each corner (Fig. 8). Quantitatively, every error metric

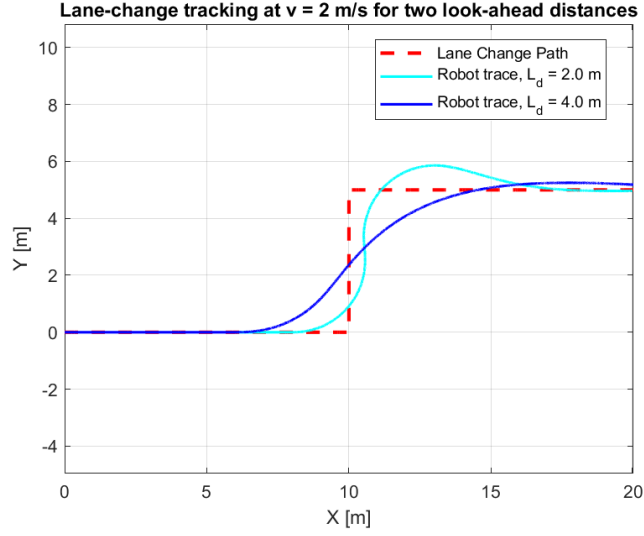


Figure 7: Lane-change trace at  $v = 2 \text{ m s}^{-1}$  with  $L_d = 2 \text{ m}$ .

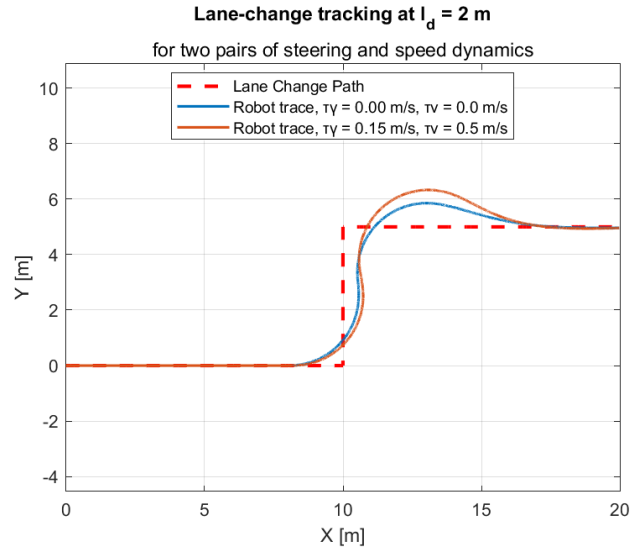


Figure 8: Lane-change trace with ideal actuation ( $\tau_\gamma = \tau_v = 0$ , blue) versus realistic lags ( $\tau_\gamma = 0.15 \text{ s}$ ,  $\tau_v = 0.5 \text{ s}$ , red) at the same  $L_d = 2 \text{ m}$ .



Table 2: CTE statistics for ideal vs. lagged actuators.

	Ideal	Lags
Mean / m	0.1819	0.2675
Max / m	0.8546	1.3308
95th perc. / m	0.7533	1.2192
RMS / m	0.3176	0.4737

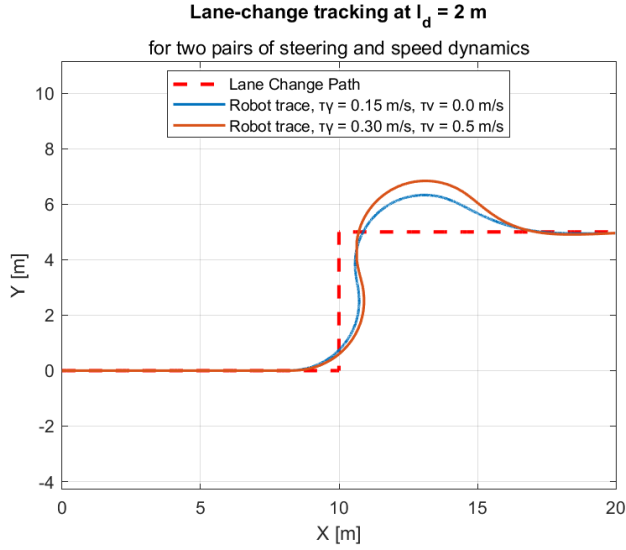


Figure 9: Effect of doubling the steering lag to  $\tau_\gamma = 0.30$  s ( $\tau_v = 0$ ).

in Table 2 grows by 45%–56%; the peak error rises to 1.33 m, reflecting the vehicle’s inability to reach  $\gamma_{\max}$  quickly enough to follow the sharp  $90^\circ$  turn.

## D.2 Doubling the steering lag

Doubling  $\tau_\gamma$  amplifies the corner overshoot (Fig. 9). RMS error increases by 43 %, and the worst-case deviation reaches 1.84 m. The trend confirms the inverse relationship between steering bandwidth and path-tracking accuracy.

## D.3 Tightening the steering-angle limit

Restricting  $\gamma_{\max}$  forces the tractor to *cut* the inside of both turns (Fig. 10). With  $\gamma_{\max} = 25^\circ$ , the peak cross-track error exceeds 2.5 m and the RMS error almost triples relative to the nominal  $45^\circ$  limit (Table 4). Relaxing the constraint to  $35^\circ$  halves the maximum error yet still doubles the mean and RMS compared with the unrestricted case. Hence, adequate steering range is crucial for accurate lane-change manoeuvres.

Table 3: CTE statistics versus steering-lag magnitude.

	$\tau_\gamma = 0.15$ s	$\tau_\gamma = 0.30$ s
Mean / m	0.2675	0.3849
Max / m	1.3308	1.8410
95th perc. / m	1.2192	1.7293
RMS / m	0.4737	0.6762

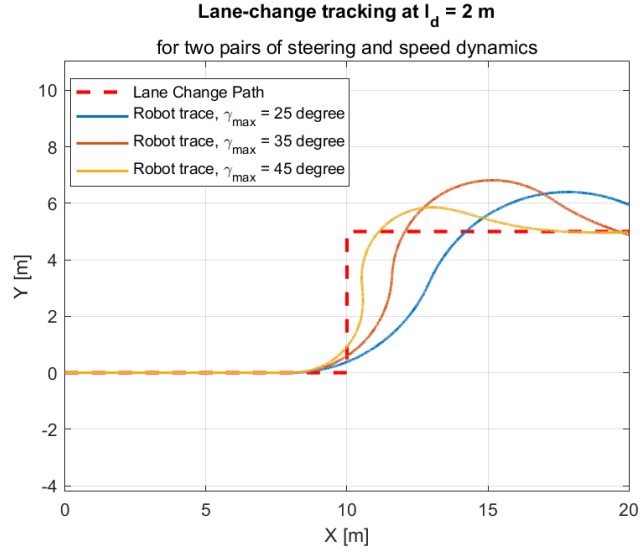


Figure 10: Lane-change trace for  $\gamma_{\max} \in \{25^\circ, 35^\circ, 45^\circ\}$ .

**Key takeaways** \* **Actuator lags** shift the closed-loop poles toward the right-half plane, degrading both accuracy and peak error proportionally to  $\tau_\gamma$ . \* **Steering limits** cap the achievable curvature. Below  $\gamma_{\max} \approx 35^\circ$  the vehicle cannot negotiate the 5 m lateral displacement without significant path deviation. \* When both effects combine (large  $\tau_\gamma$  and small  $\gamma_{\max}$ ) the controller may fail to converge, suggesting the need for a larger  $L_d$ , a slower speed, or a more advanced preview controller such as Stanley or MPC.

Table 4: CTE statistics versus steering-angle limit.

	25°	35°	45°
Mean / m	0.4980	0.4813	0.1819
Max / m	2.5283	1.8121	0.8546
95th perc. / m	1.9924	1.7336	0.7533
RMS / m	0.8537	0.7922	0.3176