

#### § 13.5 卡氏定理

意大利工程师 阿尔伯托·卡斯蒂利亚诺在其博士论文里导出了用于计算弹性构件力和位移的两个定理:

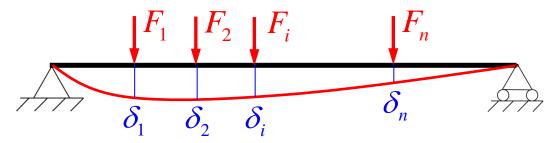
(Alberto Castigliano, 1847~1884)

卡氏第一定理(1873年)

卡氏第二定理(1873年)



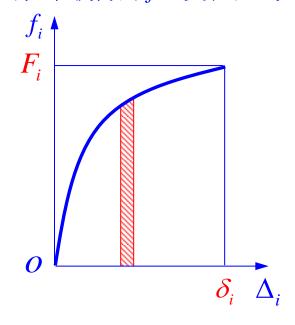
#### I. 卡氏第一定理



图示梁(材料为线性,也可为非线性) 作用n个集中载荷 $F_i$  (i=1, 2, ...n), 相应位移为 $\delta_i$  (i=1, 2, ...n)

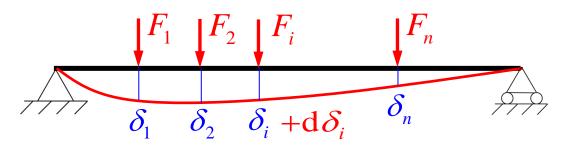
相应位移为
$$\delta_i$$
  $(i=1,2,\ldots n)$  梁内的应变能:  $V_{\varepsilon} = W = \sum_{i=1}^n \int_0^{\delta_i} f_i d\Delta_i$ 

第i个载荷的 f-Δ 变化曲线



梁内的应变能是最终位移 $\delta_i$  (i=1,2,...,n) 的函数,即

$$V_{\varepsilon} = V_{\varepsilon}(\delta_1, \delta_2 \cdots \delta_n)$$



若与第i个载荷相应的位移有一微小增量 $d\delta_i$ ,则梁内应变能的变化 $dV_c$ 应写作:

$$dV_{\varepsilon} = \frac{\partial V_{\varepsilon}(\delta_1, \delta_2 \cdots \delta_n)}{\partial \delta_i} \cdot d\delta_i$$

因为只有第i个载荷相应的位移有一微小增量 $d\delta_i$ ,其余载荷相应的位移保持不变,则外力功的变化可写作**:** 

$$dW = F_i \cdot d\delta_i$$

$$\begin{split} \mathrm{d}V_{\varepsilon} &= \frac{\partial V_{\varepsilon}(\delta_1, \delta_2 \cdots \delta_n)}{\partial \delta_i} \cdot \mathrm{d}\delta_i \quad \frac{\mathbf{功能原理}}{\mathbf{d}W} = F_i \cdot \mathrm{d}\delta_i \\ F_i &= \frac{\partial V_{\varepsilon}(\delta_1, \delta_2 \cdots \delta_n)}{\partial \delta_i} \quad \text{卡氏第一定理} \end{split}$$

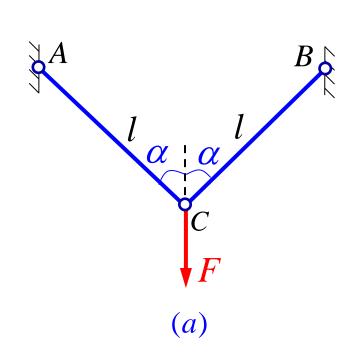
弹性杆件的应变能对任一位移 $\delta_i$ 的偏导数,等于该位移方向上作用的载荷 $F_i$ 。

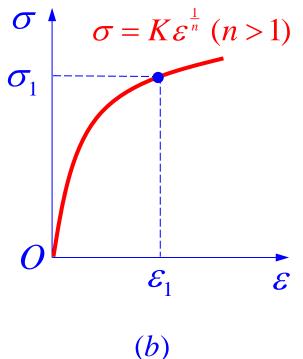
卡氏定理适用于一切受力状态下的弹性杆件。

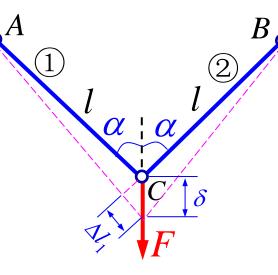
卡氏定理对非线性弹性杆件也适用。

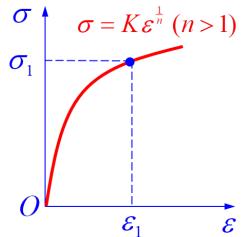
 $F_i$ : 广义力: 一个力、一个力偶、一对力、一对力偶  $\delta_i$ : 广义位移: 线位移、角位移、相对线位移、相对角位移

例1 由两根完全相同的非线性弹性杆铰接而成的结构,如图所示。在节点 C作用一铅垂力F。若两杆的长度均为l,横截面面积均为A。试求节点 C的铅垂位移 $\delta$ 。









#### 解:

节点C的铅垂位移δ与两杆的伸长量之间的关系为

$$\Delta l_1 = \Delta l_2 = \delta \cos \alpha$$

两杆都是均匀变形,则两杆的纵向线应变为

$$\varepsilon_1 = \varepsilon_2 = \frac{\Delta l_1}{l} = \frac{\delta}{l} \cos \alpha$$

两杆的应变能密度为

$$v_{\varepsilon_1} = v_{\varepsilon_2} = \int_0^{\varepsilon_1} \sigma d\varepsilon = \int_0^{\varepsilon_1} K \varepsilon^{\frac{1}{n}} d\varepsilon = \frac{n}{n+1} K(\varepsilon_1)^{\frac{n+1}{n}}$$

结构的应变能  $V_{\varepsilon} = 2v_{\varepsilon_1}Al = 2Al\frac{n}{n+1}K\left(\frac{\delta}{l}\cos\alpha\right)^{\frac{n+1}{n}}$ 由卡氏第一定理,有

$$\frac{\partial V_{\varepsilon}}{\partial \delta} = F \longrightarrow \delta = \frac{1}{2^n} \left(\frac{F}{AK}\right)^n \frac{l}{\cos^{n+1}\alpha}$$

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#### 另解:

$$\delta = \frac{1}{2^n} \left( \frac{F}{AK} \right)^n \frac{l}{\cos^{n+1} \alpha}$$

$$F_{AC}\sin\alpha - F_{BC}\sin\alpha = 0$$

$$F_{AC}\cos\alpha + F_{BC}\cos\alpha = F$$

$$F_{AC} = F_{BC} = \frac{F}{2\cos\alpha}$$

$$F_{AC} = F_{BC} = \frac{F}{2\cos\alpha}$$
  $\sigma_1 = \sigma_2 = \frac{F_{AC}}{A} = \frac{F}{2\cos\alpha \cdot A}$ 

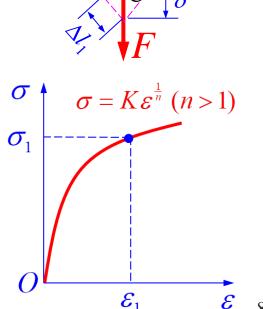
$$\sigma = K\varepsilon^{\frac{1}{n}} \ (n > 1) \Longrightarrow \varepsilon = \left(\frac{\sigma}{K}\right)^n$$

$$\varepsilon_1 = \varepsilon_2 = \frac{\Delta l_1}{l}$$

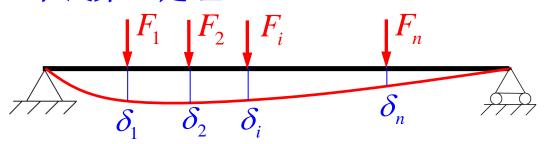
$$\varepsilon_1 = \varepsilon_2 = \left(\frac{\sigma_1}{K}\right)^n = \left(\frac{F}{2\cos\alpha \cdot AK}\right)^n \quad \sigma_1$$

$$\Delta l_1 = \Delta l_2 = \left(\frac{F}{2\cos\alpha \cdot AK}\right)^n l$$

$$\Delta l_1 = \Delta l_2 = \delta \cos \alpha \longrightarrow \delta = \frac{\Delta l_1}{\cos \alpha} = \frac{1}{2^n} \left(\frac{F}{AK}\right)^n \frac{l}{\cos^{n+1} \alpha}$$



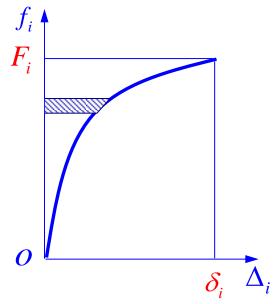
#### II. 卡氏第二定理



图示梁(材料为线性,也可为非线性) 作用n个集中载荷 $F_i$  (i=1, 2, ...n),相应 位移为 $\delta_i$  (i=1, 2, ...n),

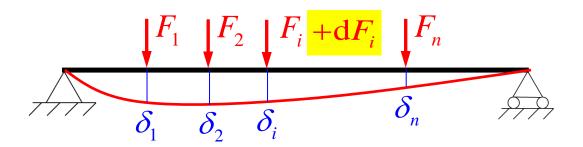
梁内的余能: 
$$V_c = W_c = \sum_{i=1}^n \int_0^{F_i} \Delta_i df_i$$

第i个载荷的 f- $\Delta$  变化曲线



最终梁内的余能应是关于 $F_i$  (i=1,2...n) 的函数,即

$$V_c = V_c(F_1, F_2 \cdots F_n)$$



若第i个载荷有一微小增量 $dF_i$ ,则梁内余能的变化量 $dV_c$ 应写作:

$$dV_c = \frac{\partial V_c(F_1, F_2 \cdots F_n)}{\partial F_i} \cdot dF_i$$

因为只有第i个载荷有一微小改变量 $dF_i$ ,其余载荷均保持不变,则外力总余功的变化可写作:

$$dW_c = \delta_i \cdot dF_i$$

$$dV_c = \frac{\partial V_c(F_1, F_2 \cdots F_n)}{\partial F_i} \cdot dF_i \qquad dW_c = \delta_i \cdot dF_i$$
 
$$\delta_i = \frac{\partial V_c(F_1, F_2 \cdots F_n)}{\partial F_i} \qquad$$
 余能定理

弹性杆件的余能对任一载荷 $F_i$ 的偏导数,等于 $F_i$ 作用点沿 $F_i$ 方向的位移 $\delta_i$ 。

余能定理适用于一切受力状态下的弹性杆件。

余能定理对非线性弹性杆件也适用。

$$\delta_i = \frac{\partial V_c(F_1, F_2 \cdots F_n)}{\partial F_i}$$
 余能定理

 $F_i$ : 广义力: 一个力、一个力偶、一对力、一对力(偶)

 $\delta_i$ :广义位移:线位移、角位移、相对线位移、相对角位移 对于线弹性杆件或杆系,由于力与位移成正比,杆内的应变能 V。数值上等于余能V。,则余能定理此时可改写为:

$$\delta_i = \frac{\partial V_{\varepsilon}(F_1, F_2 \cdots F_n)}{\partial F_{\varepsilon}}$$
 卡氏第二定理

 $\delta_i = \frac{\partial V_{\varepsilon}(F_1, F_2 \cdots F_n)}{\partial F_i} \quad \text{卡氏第二定理}$  线弹性杆件的应变能对任一载荷 $F_i$ 的偏导数,等于 $F_i$ 作用点沿  $F_i$ 方向的位移 $\delta_i$ 。

卡氏第二定理只适用于线性弹性情况。

例2 求悬臂梁自由端作用集中力F 时自由端处的挠度和转角。 (梁的弯曲刚度为EI,长为l)。

$$rac{F}{x}$$

解: (1) 求自由端处的挠度  $w_c = \frac{\partial V_{\varepsilon}}{\partial F}$ 

$$V_{\varepsilon} = \int_{0}^{l} \frac{M^{2}(x)}{2EI} dx = \int_{0}^{l} \frac{(-Fx)^{2}}{2EI} dx = \frac{F^{2}l^{3}}{6EI}$$

$$\overset{\bullet}{o} \qquad w_c = \frac{\partial V_{\varepsilon}}{\partial F} = \frac{Fl^3}{3EI}$$

建议 
$$w_c = \frac{\partial V_{\varepsilon}}{\partial F}$$
  $\longrightarrow w_c = \int_0^l \frac{M(x)}{EI} \cdot \frac{\partial M(x)}{\partial F} dx$   $\longrightarrow w_c = \int_0^l \frac{(-Fx)}{EI} \cdot (-x) dx$   $V_{\varepsilon} = \int_0^l \frac{M^2(x)}{2EI} dx$  结果为正,表示位移方向  $= \frac{F}{EI} \cdot \frac{l^3}{3} = \frac{Fl^3}{3EI}$ 

与F的方向一致!

M 解: (2) 求自由端处的转角 M 假想在自由端作用一力矩M

$$\theta = \frac{\partial V_{\varepsilon}}{\partial M} \bigg|_{M=0}$$

$$V_{\varepsilon} = \int_0^l \frac{M_{AB}^2(x)}{2EI} dx \qquad M_{AB}(x) = -(Fx + M)$$

|接下来先完成求导工作, 不要先去做积分!

$$= \int_0^l \frac{-(Fx+M)}{EI} \cdot (-1) \, dx \bigg|_{M=0} = \int_0^l \frac{Fx}{EI} dx = \frac{Fl^2}{2EI}$$

思考 若假想在自由端作用一力矩*M* 的方向与前面相反,结果有何变化?

$$V_{\varepsilon} = \int_{0}^{l} \frac{M_{AB}^{2}(x)}{2EI} dx = \int_{0}^{l} \frac{(M - Fx)^{2}}{2EI} dx$$

$$\theta = \frac{\partial V_{\varepsilon}}{\partial M} \bigg|_{M=0} = \int_{0}^{l} \frac{(M - Fx)}{EI} \cdot 1 \cdot dx \bigg|_{M=0} = -\int_{0}^{l} \frac{Fx}{EI} dx = -\frac{Fl^{2}}{2EI}$$
"负号"说明  $\theta$  的转向与所加弯矩M反向

#### 例3 求图示结构结点A的铅垂位移和水平位移。

#### 应用卡氏第二定理进行求解

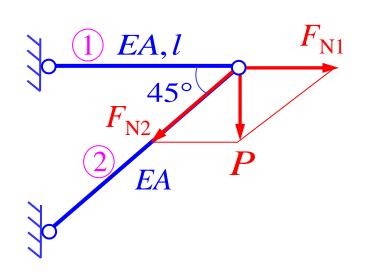
解: 先求铅垂位移

#### 两杆的内力(将力直接分解)

$$F_{\rm N1} = P(\stackrel{.}{\cancel{2}}), \qquad F_{\rm N2} = \sqrt{2}P(\stackrel{.}{\cancel{2}})$$

$$V_{\varepsilon} = \frac{P^2 l}{2EA} + \frac{(\sqrt{2}P)^2 \sqrt{2}l}{2EA} = (2\sqrt{2} + 1)\frac{P^2 l}{2EA}$$

$$\Delta A_y = \frac{\partial V_{\varepsilon}}{\partial P} = (2\sqrt{2} + 1)\frac{Pl}{EA}$$



# 解: 求水平方向位移 $\Delta A_x = \frac{\partial V_{\varepsilon}}{\partial F}$ 两杆的内力

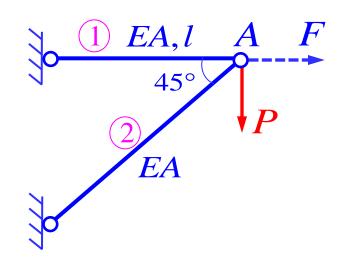
$$F_{\text{N1}} = P + F,$$
  $F_{\text{N2}} = \sqrt{2}P$   $(P + F)^2 I$   $(\sqrt{2}P)^2 \sqrt{2}I$ 

$$V_{\varepsilon} = \frac{(P+F)^2 l}{2EA} + \frac{(\sqrt{2}P)^2 \sqrt{2}l}{2EA}$$

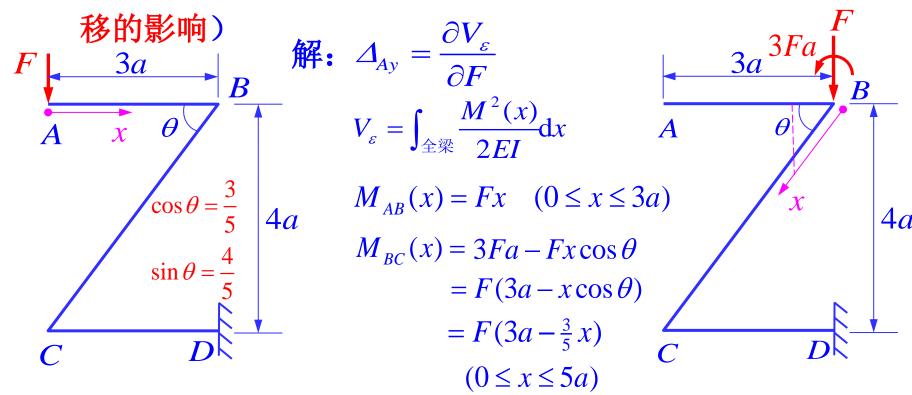
$$\Delta A_x = \frac{\partial V_{\varepsilon}}{\partial F} \Big|_{F=0} = \frac{(P+F)l}{EA} \Big|_{F=0} + 0 = \frac{Pl}{EA}$$

先求导

代入
$$F=0$$



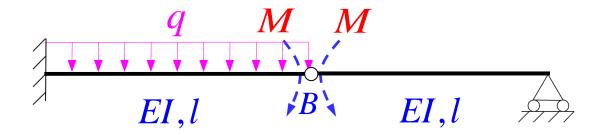
## 例4 求Z字型平面刚架受铅垂方向集中力F作用时A点的铅垂位移,已知各杆的弯曲刚度均为EI。(不计轴力和剪力对位



 $= \int_0^{3a} \frac{Fx}{FI} \cdot x dx + \int_0^{5a} \frac{3Fa - \frac{3}{5}Fx}{FI} \cdot \frac{3a - \frac{3}{5}x}{\partial F} dx + \int_0^{3a} \frac{Fx}{FI} \cdot x dx = \frac{33Fa^3}{FI}$ 

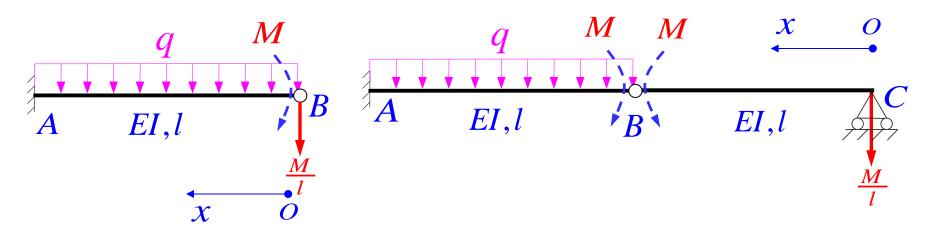
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#### 例5 求铰B两侧截面的相对转角(各杆的弯曲刚度为EI)。



#### 解: 在铰B处施加一对力矩M,则

$$\theta_B = \frac{\partial V_{\varepsilon}}{\partial M} \bigg|_{M = 0}$$



$$M_{AB}(x) = -\frac{M}{l} x - M - \frac{1}{2} qx^2$$

$$M_{BC}(x) = -\frac{M}{l} x$$

$$V_{\varepsilon} = \int_0^l \frac{M_{AB}^2(x)}{2EI} dx + \int_0^l \frac{M_{BC}^2(x)}{2EI} dx$$

$$\frac{\partial V_{\varepsilon}}{\partial M} = \int_{0}^{l} \frac{M_{AB}(x)}{EI} \cdot \frac{\partial M_{AB}(x)}{\partial M} dx + \int_{0}^{l} \frac{M_{BC}(x)}{EI} \cdot \frac{\partial M_{BC}(x)}{\partial M} dx$$

$$M_{AB}(x) = -\frac{M}{l} x - M - \frac{1}{2} qx^{2} \qquad M_{BC}(x) = -\frac{M}{l} x$$

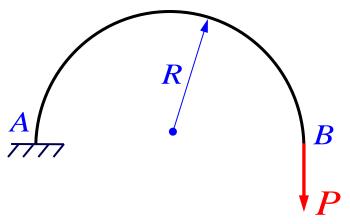
$$\frac{\partial M_{AB}(x)}{\partial M} = -\left(1 + \frac{x}{l}\right) \qquad \frac{\partial M_{BC}(x)}{\partial M} = -\frac{x}{l}$$

$$\frac{\partial V_{\varepsilon}}{\partial M} = \int_{0}^{l} \frac{M_{AB}(x)}{EI} \cdot \frac{\partial M_{AB}(x)}{\partial M} dx + \int_{0}^{l} \frac{M_{BC}(x)}{EI} \cdot \frac{\partial M_{BC}(x)}{\partial M} dx$$

$$\theta_{B} = \frac{\partial V_{\varepsilon}}{\partial M} \bigg|_{M=0} = \frac{1}{EI} \int_{0}^{l} \left(-\frac{1}{2} qx^{2}\right) \cdot \left[-\left(1 + \frac{x}{l}\right)\right] dx + \frac{1}{EI} \int_{0}^{l} 0 \cdot \left(-\frac{x}{l}\right) dx$$

$$= \frac{q}{2EI} \int_{0}^{l} \left(x^{2} + \frac{x^{3}}{l}\right) dx = \frac{q}{2EI} \left(\frac{l^{3}}{3} + \frac{l^{3}}{4}\right) = \frac{7ql^{3}}{24EI}$$

### 例6 求图示半圆形曲杆B点的水平位移和铅垂位移。(不计轴力和剪力对位移的影响)。



#### 解: (1) 求铅垂位移

曲杆的内力:  $F_{N}, F_{s}, M$ 

$$F_{\rm N}(\theta) = P\cos\theta$$

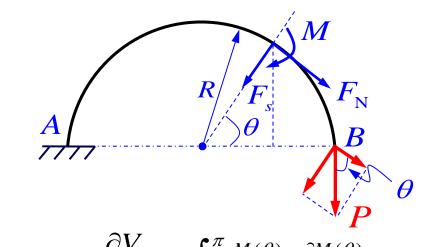
$$F_s(\theta) = P \sin \theta$$

$$M(\theta) = P(R - R\cos\theta)$$

直杆: 
$$V_{\varepsilon} = \int \frac{M^2(x)}{2EI} dx$$

曲杆: 
$$V_{\varepsilon}(s) = \int \frac{M^2(s)}{2EI} ds$$
 曲线坐标形式

$$V_{\varepsilon}(\theta) = \int \frac{M^{2}(\theta)}{2EI} R d\theta$$
 极坐标形式



$$\Delta_{By} = \frac{\partial V_{\varepsilon}}{\partial P} = \int_{0}^{\pi} \frac{M(\theta)}{EI} \cdot \frac{\partial M(\theta)}{\partial P} \cdot R d\theta$$
$$= \frac{1}{EI} \int_{0}^{\pi} P[R(1 - \cos \theta)]^{2} R d\theta$$

$$=\frac{3\pi PR^3}{2EI}\left(\downarrow\right)$$

#### 解: (2) 求水平位移

#### 虚加一水平力F

$$M(\theta) = FR \sin \theta - P(R - R \cos \theta)$$

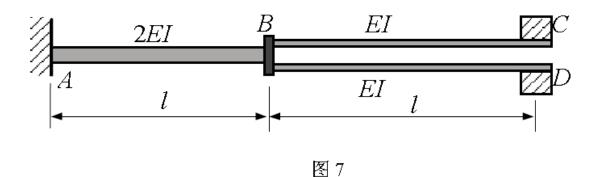
$$\Delta_{Bx} = \frac{\partial V_{\varepsilon}}{\partial F} \bigg|_{F=0} = \int_{0}^{\pi} \frac{M(\theta)}{EI} \cdot \frac{\partial M(\theta)}{\partial F} \cdot R d\theta \bigg|_{F=0}$$

$$= \frac{1}{EI} \int_0^{\pi} \left[ -PR(1 - \cos \theta) \right] \cdot R \sin \theta \cdot R d\theta$$

$$= -\frac{PR^3}{EI} \int_0^{\pi} (\sin \theta - \sin \theta \cos \theta) d\theta = -\frac{2\pi PR^3}{EI} (\leftarrow)$$

### 第十一届全国周培源大学生力学竞赛(个人赛)试题

一种能量收集装置,可简化为图 7 所示悬臂梁模型。梁 AB 长 I,弯曲刚度为 2EI;梁 BC、BD 长均为 I,弯曲刚度均为 EI。梁 AB 与梁 BC、BD 通过刚节点 B 连接,三梁均处于水平位置。梁和刚节点 B 的重量均不计。梁 BC、BD 端部均固定有重量为 W 的物块,该两梁之间有小间隙。则梁端 D 的挠度与物块重量之比  $f_D/W=$  (\_\_\_\_\_\_\_\_\_)。



$$\frac{f_D}{W} = \frac{8l^3}{3EI}$$

#### 解:【卡氏第二定理】:

$$\left. f_D = \frac{\partial V_{\varepsilon}}{\partial P} \right|_{P = W}$$

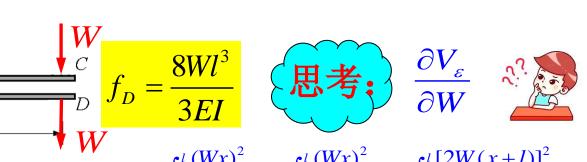
$$V_{\varepsilon} = \int_0^l \frac{(Wx)^2}{2EI} dx + \int_0^l \frac{(Px)^2}{2EI} dx$$

$$f_{D} = 0 + \int_{0}^{l} \frac{(Px)x}{EI} dx \Big|_{P=W} + \int_{0}^{l} \frac{[(W+P)(x+l)](x+l)}{2EI} dx \Big|_{P=W} + \int_{0}^{l} \frac{[(W+P)(x+l)]^{2}}{2 \times 2EI} dx$$

$$c_{l} Wx^{2} = c_{l} [2W(x+l)]^{2} = Wl^{3} = W(l^{3}) = 8Wl^{3}$$

$$= 0 + \int_0^l \frac{Wx^2}{EI} dx + \int_0^l \frac{[2W(x+l)]^2}{2EI} dx = \frac{Wl^3}{3EI} + \frac{W}{EI} \left(\frac{l^3}{3} + l^3 + l^3\right) = \frac{8Wl^3}{3EI}$$







$$V_{\varepsilon} = \int_{0}^{l} \frac{(Wx)^{2}}{2EI} dx + \int_{0}^{l} \frac{(Wx)^{2}}{2EI} dx + \int_{0}^{l} \frac{[2W(x+l)]^{2}}{2 \times 2EI} dx$$

$$\frac{\partial V_{\varepsilon}}{\partial W} = \int_0^l \frac{(Wx)x}{EI} dx + \int_0^l \frac{(Wx)x}{EI} dx + \int_0^l \frac{[(2W)(x+l)] \times 2(x+l)}{2EI} dx$$
$$= \int_0^l \frac{Wx^2}{EI} dx + \int_0^l \frac{Wx^2}{EI} dx + \int_0^l \frac{[4W(x+l)]^2}{2EI} dx$$

$$= 2 \times \frac{Wl^{3}}{3EI} + \frac{2W}{EI} \left( \frac{l^{3}}{3} + l^{3} + l^{3} \right) = 2 \times \frac{8Wl^{3}}{3EI} = \frac{16Wl^{3}}{3EI} = 2f_{D}$$

$$\frac{\partial V_{\varepsilon}}{\partial W} = f_{C} + f_{D}$$

$$f_{C} = f_{D}$$

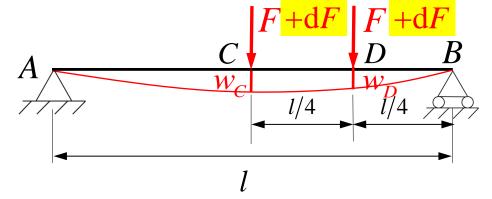
$$f_{D} = \frac{1}{2} \frac{\partial V_{\varepsilon}}{\partial W}$$







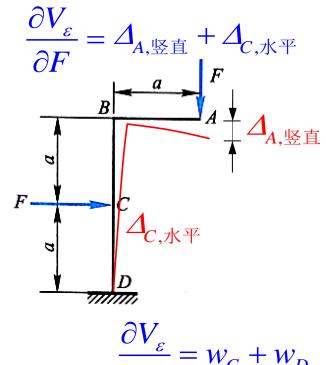
$$\frac{\partial V_{\varepsilon}}{\partial F} = ?$$



#### 线弹性问题: $V_{c}(F) = V_{\varepsilon}(F)$

$$dV_{c}(F) = dV_{\varepsilon}(F) = \frac{\partial V_{\varepsilon}(F)}{\partial F} dF$$

$$dV_{c}(F) = w_{C}dF + w_{D}dF = (w_{C} + w_{D})dF$$



$$\frac{\partial V_{\varepsilon}}{\partial F} = w_C + w_D$$
$$\frac{\partial V_{\varepsilon}(F)}{\partial F} dF = (w_C + w_D) dF$$

## 翻翻大家

作业 (第二册)

P125-126: 13.6, 13.7(b), 13.8(b)

对应第6版的题号 P117-118: 13.6, 13.7(b), 13.8(b)

下次课讲 虚功原理 单位载荷法