Revisit to the theoretical analysis of a classical piezoelectric cantilever energy harvester

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1 Introduction

The soaring depletion of fossil fuels and the growing awareness of environmental protection renders the challenge of renewable and sustainable energy sources for scientists and engineers. Apart from nuclear energy, wind energy, and solar energy, ambient energy harvesting has been proposed as an alternative to batteries for more than twenty years. Numerous principles, mechanisms, implementations, and applications have been put forward to deepen the understanding of energy harvesting, to find the routine for the development of new devices, to apply the devices to new situations. Among all these devices, piezoelectric energy harvesting devices have attracted much attention due to their simplicity of structure and superiority of performance.

Generally a piezoelectric energy harvesting device consists of some piezoelectric elements and some vibration transduction structure for the hosting of these elements. A key problem in the research of piezoelectric element based energy harvesting is to improve the output of those devices. A key point is to understand the mechanics and coupling behind the devices.

2 Outline of the paper

The outline of the paper should be as follows:

- To obtain the closed form solution of the CPEH problem using the harmonic balance method
- Analyze the dependence of relative displacement function $u(z;\delta)$
- Tackling the dependence of output index χ_p and output measures \tilde{V}_p , \tilde{I}_p , and \tilde{P}_p upon the electromechanical coupling factor δ and base excitation frequency f_b
- Derive the asymptotic expansion of the CPEH problem for the displacement function $u(z;\delta)$ and the output index χ_p
- Explore the approximation error of the asymptotic expansion and provide some clues to improve the performance

3 Summary of the interested equations

The dynamic equations for a typical piezoelectric composite cantilever beam is

$$B_p \frac{\partial^4 w(x,t)}{\partial x^4} + m_p \frac{\partial^2 w(x,t)}{\partial t^2} = 0, \tag{1}$$

where B_p is the equivalent bending stiffness and m_p is the line mass density of the piezoelectric cantilever beam. If the piezoelectric elements attached to the cantilever beam is connected to an external electrical load R_l , we have

$$\frac{dQ_p(t)}{dt} + \frac{V_p(t)}{R_l} = 0. (2)$$

For the underlying physics, we have the following constitutive equations

$$M_p(x,t) = B_p \frac{\partial^2 w(x,t)}{\partial x^2} - e_p V_p(t),$$

$$q_p(x,t) = e_p \frac{\partial^2 w(x,t)}{\partial x^2} + \varepsilon_p V_p(t),$$
(3)

or equivalently,

$$\begin{cases}
M_p(x,t) = B_p \frac{\partial^2 w(x,t)}{\partial x^2} - e_p V_p(t), \\
Q_p(x,t) = e_p \left[\frac{\partial w(x,t)}{\partial x} \right]_0^{l_p} + C_p V_p(t).
\end{cases}$$
(4)

One end of the cantilever beam is fixed while the other end is free. So the boundary conditions are

$$\begin{cases} w(0,t) = w_b(t), \\ \frac{\partial w(0,t)}{\partial x} = 0, \end{cases}$$
 (5)

and

$$\begin{cases}
M_p(l_p, t) = B_p \frac{\partial^2 w(l_p, t)}{\partial x^2} - e_p V_p(t) = 0, \\
N_p(l_p, t) = \frac{\partial M_p(l_p, t)}{\partial x} = B_p \frac{\partial^3 w(l_p, t)}{\partial x^3} = 0.
\end{cases}$$
(6)

4 Theoretical solution to the problem

In the classical energy harvesting applications, the cantilever beam is subject to a periodical base excitation $w_b(t)$. Thus the dynamic response of the cantilever beam is decomposed as

$$w(x,t) = w_b(t) + w_{rel}(x,t), \tag{7}$$

where $w_{rel}(x,t)$ is the relative displacement function of the cantilever beam. In this way, the system is converted into

$$B_{p}\frac{\partial^{4}w_{rel}(x,t)}{\partial x^{4}} + m_{p}\frac{\partial^{2}w_{rel}(x,t)}{\partial t^{2}} = -m_{p}\frac{\partial^{2}w_{b}(t)}{\partial t^{2}},$$
(8)

$$e_p \left[\frac{\partial^2 w_{rel}(x,t)}{\partial x \partial t} \right] \Big|_0^{l_p} + C_p \frac{dV_p(t)}{dt} + \frac{V_p(t)}{R_l} = 0.$$
 (9)

$$\begin{cases} w_{rel}(0,t) = 0, \\ \frac{\partial w_{rel}(0,t)}{\partial x} = 0, \end{cases}$$
 (10)

and

$$\begin{cases}
B_p \frac{\partial^2 w_{rel}(l_p, t)}{\partial x^2} - e_p V_p(t) = 0, \\
\frac{\partial^3 w_{rel}(l_p, t)}{\partial x^3} = 0.
\end{cases}$$
(11)

Considering a sinusoidal base excitation

$$w_b(t) = \eta_b e^{j\sigma_b t} \tag{12}$$

where ξ_b is usually a real vibration amplitude, the steady state solution for the above system can be reasonably set as

$$w_{rel}(x,t) = \eta_{rel}(x)e^{j\sigma_b t}, \quad V_p(t) = \tilde{V}_p e^{j\sigma_b t}, \tag{13}$$

where $\eta_{rel}(x)$ and \tilde{V}_p are complex amplitudes. Then the above system is again simplified as

$$B_p \frac{\partial^4 \eta_{rel}(x)}{\partial x^4} - m_p \sigma_b^2 \eta_{rel}(x) = m_p \sigma_b^2 \eta_b, \tag{14}$$

$$\begin{cases}
\eta_{rel}(0) = 0, \\
\frac{\partial \eta_{rel}(0)}{\partial x} = 0,
\end{cases}$$
(15)

and

$$\begin{cases}
B_p \frac{\partial^2 \eta_{rel}(l_p)}{\partial x^2} + \frac{j\sigma_b R_l}{1 + j\sigma_b C_p R_l} e_p^2 \frac{\partial \eta_{rel}(l_p)}{\partial x} = 0, \\
\frac{\partial^3 \eta_{rel}(l_p)}{\partial x^3} = 0.
\end{cases}$$
(16)

Note that here we assume a sinusoidal steady state response, which is not actually validated theoretically.

Obviously we can have the following dimensionless scheme:

$$\eta_{rel} \sim u\eta_b, \quad x \sim zl_p$$
(17)

and therefore the following dimensionless parameters

$$\sigma = \sigma_b \sqrt{\frac{m_p l_p^4}{B_p}}, \quad \beta = R_l C_p \sqrt{\frac{B_p}{m_p l_p^4}}, \quad \delta = \frac{e_p^2 l_p}{C_p B_p}. \tag{18}$$

Now, we reach the following dimensionless system of boundary value problem

$$\begin{cases} u'''' - \sigma^2 u = \sigma^2, \\ u(0) = 0, \\ u'(0) = 0, \end{cases}$$

$$u''(1) + \frac{j\beta\sigma}{1 + j\beta\sigma} \delta u'(1) = 0,$$

$$u'''(1) = 0,$$
(19)

where the prime denotes the derivative with respect to z. The analytical solution to this problem can be formulated as

$$u(z;\delta) = A_{\delta}\cos\sqrt{\sigma}z + B_{\delta}\sin\sqrt{\sigma}z + C_{\delta}\cosh\sqrt{\sigma}z + D_{\delta}\sinh\sqrt{\sigma}z - 1$$
 (20)

and hence

$$u'(z;\delta) = \sigma^{1/2} \left(-A_{\delta} \sin \sqrt{\sigma}z + B_{\delta} \cos \sqrt{\sigma}z + C_{\delta} \sinh \sqrt{\sigma}z + D_{\delta} \cosh \sqrt{\sigma}z \right),$$

$$u''(z;\delta) = \sigma \left(-A_{\delta} \cos \sqrt{\sigma}z - B_{\delta} \sin \sqrt{\sigma}z + C_{\delta} \cosh \sqrt{\sigma}z + D_{\delta} \sinh \sqrt{\sigma}z \right),$$

$$u'''(z;\delta) = \sigma^{3/2} \left(A_{\delta} \sin \sqrt{\sigma}z - B_{\delta} \cos \sqrt{\sigma}z + C_{\delta} \sinh \sqrt{\sigma}z + D_{\delta} \cosh \sqrt{\sigma}z \right).$$
(21)

The coefficients A_{δ} , B_{δ} , C_{δ} , and D_{δ} are then subject to the following linear system of equations:

$$\begin{cases}
A_{\delta} + C_{\delta} = 1, \\
B_{\delta} + D_{\delta} = 0, \\
(-A_{\delta} \cos \sqrt{\sigma} - B_{\delta} \sin \sqrt{\sigma} + C_{\delta} \cosh \sqrt{\sigma} + D_{\delta} \sinh \sqrt{\sigma}) + \\
\frac{j\beta\sqrt{\sigma}}{j\sigma\beta + 1} \delta \left(-A_{\delta} \sin \sqrt{\sigma} + B_{\delta} \cos \sqrt{\sigma} + C_{\delta} \sinh \sqrt{\sigma} + D_{\delta} \cosh \sqrt{\sigma} \right) = 0, \\
A_{\delta} \sin \sqrt{\sigma} - B_{\delta} \cos \sqrt{\sigma} + C_{\delta} \sinh \sqrt{\sigma} + D_{\delta} \cosh \sqrt{\sigma} = 0.
\end{cases} (22)$$

Analytically, we can directly obtain the solution to this problem as

$$A_{\delta} = \frac{1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma} - \sin\sqrt{\sigma}\sinh\sqrt{\sigma} + \frac{2j\beta\sqrt{\sigma}}{1+j\beta\sigma}\delta\left(\cos\sqrt{\sigma}\sinh\sqrt{\sigma}\right)}{2\left[1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma} + \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\delta\left(\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}\right)\right]},$$

$$B_{\delta} = \frac{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma} + \frac{2j\beta\sqrt{\sigma}}{1+j\beta\sigma}\delta\left(\sin\sqrt{\sigma}\sinh\sqrt{\sigma}\right)}{2\left[1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma} + \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\delta\left(\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}\right)\right]},$$

$$C_{\delta} = \frac{1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma} + \sin\sqrt{\sigma}\sinh\sqrt{\sigma} + \frac{2j\beta\sqrt{\sigma}}{1+j\beta\sigma}\delta\left(\sin\sqrt{\sigma}\cosh\sqrt{\sigma}\right)}{2\left[1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma} + \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\delta\left(\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}\right)\right]},$$

$$D_{\delta} = \frac{-\cos\sqrt{\sigma}\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}\cosh\sqrt{\sigma} - \frac{2j\beta\sqrt{\sigma}}{1+j\beta\sigma}\delta\left(\sin\sqrt{\sigma}\sinh\sqrt{\sigma}\right)}{2\left[1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma} + \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\delta\left(\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}\right)\right]}.$$

$$(23)$$

Firstly, to validate the closed form solution $u(z;\delta)$, we calculate the corresponding normalized output voltage $|\tilde{V}_p/(\sigma_b^2\xi_b)|$ and plot the results with respect to the base excitation frequency fr at different externally connected resistance R_l . The results are shown in Figure 1. According to the results, our calculations show a good agreement with those presented in the reference. [1]. Thus we have validated the model and the results and made our preparations for future analysis. It should be noted that, out results show an obvious difference from those in the literature [1] near the resonant frequencies. This could explained as follows. In our numerical calculations, the frequency range can be chosen to be as small as possible. A finer calculation grid leads to a sharper resonant peak. Another point to be noted is that at the resonant peak, the output voltage should arrive at its maximum and its derivative with resepct to fr should be zero and the second order derivative should be infinity. In this way, a small discrepency of the frequency from the resonant frequency leads to a large change in the normalized output voltage.

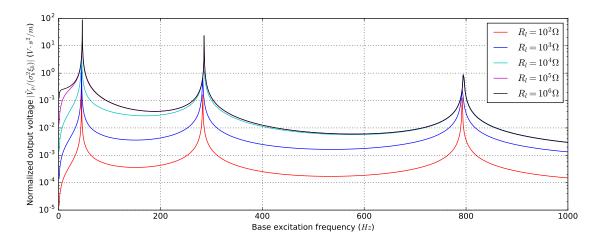


Figure 1: Validation calculation of the normalized output voltage versus base excitation frequency for a unimorph piezoelectric cantilever energy harvester based on the data from [1].

Secondly, according to equations (20) and (23), the dimensionless displacement amplitude function u(z) is totally determined by the three dimensionless parameters σ , β , and δ introduced before. Among the dimensionless parameters, σ is the dimensionless base excitation frequency, β is the dimensionless electrical resonant frequency, and δ is the dimensionless electromechanical coupling strength for the structure. As σ and β is determined by the base excitation and externally connected circuit respectively, only the parameter δ is fully determined by the structure itself. Hence we would like to investigate the influence of parameter δ upon the solution displacement function u(z). By taking different values of δ , we calculate the displacement amplitude function u(z) and plot the results in Figure 2.

It is shown in Figure 2, the parameter δ changes the function u(z) through the change of the third boundary condition (to be inserted). When δ is zero, i.e., no electromechanical coupling is present, the system degenerates to the classical elastic cantilever beam problem, whose solution is a real function. That is to say, the phase of u(z) is a constant across the whole beam (in the range of $0 \le z \le 1$). Analytical expressions for the coefficients are

$$\begin{cases} A_{\varnothing} = \frac{1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma} - \sin\sqrt{\sigma}\sinh\sqrt{\sigma}}{2\left[1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma}\right]}, \\ B_{\varnothing} = \frac{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}{2\left[1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma}\right]}, \\ C_{\varnothing} = \frac{1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma} + \sin\sqrt{\sigma}\sinh\sqrt{\sigma}}{2\left[1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma}\right]}, \\ D_{\varnothing} = \frac{-\cos\sqrt{\sigma}\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}{2\left[1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma}\right]}. \end{cases}$$
(24)

and the resulting dimensionless displacement function $u_{\varnothing}(z)$ is represented as

$$u_{\varnothing}(z) = A_{\varnothing}\cos\sqrt{\sigma}z + B_{\varnothing}\sin\sqrt{\sigma}z + C_{\varnothing}\cosh\sqrt{\sigma}z + D_{\varnothing}\sinh\sqrt{\sigma}z - 1.$$
 (25)

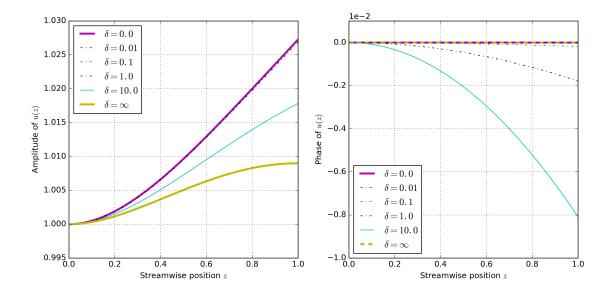


Figure 2: Amplitude and phase of the displacement function u(z) for difference values of δ

When the electromechanical coupling is extremely strong, and δ is extremely large and can be seen as ∞ in mathematical sense. In this situation, the solution $u_{\infty}(z)$ is again real without any phase difference in the z direction. The coefficients can be analytically expressed as

$$\begin{cases} A_{\infty} = \frac{\cos\sqrt{\sigma}\sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}, \\ B_{\infty} = \frac{\sin\sqrt{\sigma}\sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}, \\ C_{\infty} = \frac{\sin\sqrt{\sigma}\cosh\sqrt{\sigma}}{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}, \\ D_{\infty} = \frac{-\sin\sqrt{\sigma}\sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}. \end{cases}$$
(26)

and hence the dimensionless displacement function $u_{\infty}(z)$ is

$$u_{\infty}(z) = A_{\infty} \cos \sqrt{\sigma}z + B_{\infty} \sin \sqrt{\sigma}z + C_{\infty} \cosh \sqrt{\sigma}z + D_{\infty} \sinh \sqrt{\sigma}z - 1. \tag{27}$$

While a finite non-zero electromechanical coupling factor δ is present, which is expected in most applications, the resulting dimensionless displacement function u(z) has varying magnitude and phase along the stream-wise direction or z direction. Nevertheless, it is seen from the right panel of Figure 2 that for different values of δ , the phase change of u(z) is very small in the z direction, actually in the order 10^{-2} .

To make it more clear, we plot the phase of u(z) at z=1 versus different values of δ in Figure 3. It is clear that with the increase of δ , amplitude of the end displanment (z=1) of the beam |u(z)| decreases, while its phase reaches a minimum at around $\delta=10$. This also explains the fact expressed in Figure 2 that the amplitude of displacement function $u_{\delta}(z)$ with $0 < \delta < \infty$ is always between that of $u_{\varnothing}(z)$ and $u_{\infty}(z)$.

As for the output voltage $V_p(t)$, output current $I_p(t)$, and output power $P_p(t)$ for the classical piezoelectric cantilever energy harvester, their corresponding complex amplitudes \tilde{V}_p , \tilde{I}_p , and \tilde{P}_p

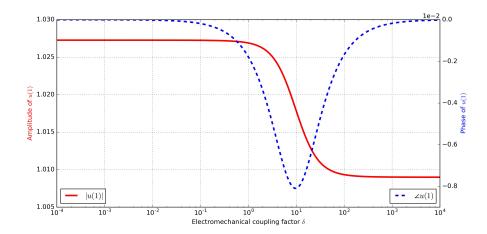


Figure 3: Amplitude and phase of the displacement function u(z) at the position z=1 versus electromechanical coupling factor δ .

can be formulated as

$$\begin{cases}
\tilde{V}_{p} = -\frac{j\sigma\beta}{j\sigma\beta+1} \frac{\eta_{b}}{l_{p}} \frac{e_{p}}{C_{p}} u'(1), \\
= -\frac{j\sigma\beta}{j\sigma\beta+1} \frac{\eta_{b}}{l_{p}} \frac{e_{p}}{C_{p}} \sigma^{1/2} \left(-A_{\delta} \sin \sqrt{\sigma} + B_{\delta} \cos \sqrt{\sigma} + C_{\delta} \sinh \sqrt{\sigma} + D_{\delta} \cosh \sqrt{\sigma} \right) \\
= -\frac{j\sigma\beta}{j\sigma\beta+1} \frac{\eta_{b}}{l_{p}} \frac{e_{p}}{C_{p}} \frac{\sqrt{\sigma} \left(\sinh \sqrt{\sigma} - \sin \sqrt{\sigma} \right)}{1 + \cos \sqrt{\sigma} \cosh \sqrt{\sigma} + \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma} \delta \left(\cos \sqrt{\sigma} \sinh \sqrt{\sigma} + \sin \sqrt{\sigma} \cosh \sqrt{\sigma} \right)} \\
= -\frac{j\sigma\beta}{j\sigma\beta+1} \left(\frac{\eta_{b}}{l_{p}} \right) \left(\frac{e_{p}}{C_{p}} \right) \chi_{p}, \\
\tilde{I}_{p} = \tilde{V}_{p}/R_{l} = -\frac{j\sigma\beta}{j\sigma\beta+1} \left(\frac{\eta_{b}}{l_{p}} \right) \left(\frac{e_{p}}{C_{p}R_{l}} \right) \chi_{p}, \\
\tilde{P}_{p} = \tilde{V}_{p}^{2}/R_{l} = \left(\frac{\eta_{b}}{l_{p}} \right)^{2} \left(\frac{e_{p}}{C_{p}} \right) \left(\frac{j\sigma\beta}{j\sigma\beta+1} \right)^{2} \chi_{p}^{2},
\end{cases}$$
which we have used the notations that

in which we have used the notations that

$$\chi_p = u_1'(1) = \frac{\sqrt{\sigma} \left(\sinh \sqrt{\sigma} - \sin \sqrt{\sigma}\right)}{1 + \cos \sqrt{\sigma} \cosh \sqrt{\sigma} + \frac{j\beta\sqrt{\sigma}}{1 + j\beta\sigma} \delta \left(\cos \sqrt{\sigma} \sinh \sqrt{\sigma} + \sin \sqrt{\sigma} \cosh \sqrt{\sigma}\right)}.$$
 (29)

Clearly, The three output measures V_p , I_p , and P_p are heavily dependent on another dimensionless parameter $r_d = \eta_b/l_p$. Formally, both \tilde{V}_p and \tilde{I}_p depend lineary upon r_d , while \tilde{P}_p shows a quadratic dependence on r_d . The only dependence upon δ is introduced in χ_p . However, it should be noted that the parameter δ relies on e_p , l_p , C_p , and B_p , while the three measures V_p , I_p , and P_p are dimensional values and depend on e_p , σ_b , and R_l . As a result, the change of parameter δ results in the change of reference voltage e_p/C_p , reference current $e_p/(C_pR_l)$, and reference power $(e_p/C_p)[e_p/(C_pR_l)]$, and therefore the corresponding values of \tilde{V}_p , \tilde{I}_p , and \tilde{P}_p . Hence, we may establish a bijective relation between δ and e_p , and relate the change of δ to that of e_p . In this way, we calculate the output measures at different values of δ and plot their amplitudes in Figure 4.

It is seen from Figure 4 that all the three measures show a maximum peak with the increase of δ at the approximate value of $\delta = 10$. When δ is small, or equivalently, e_p is small, amplitude of the three output measures \tilde{V}_p , \tilde{I}_p , and \tilde{P}_p increase with the increase of δ . Then after the critical value of δ , a further increase of δ causes the decrease of output measures. Thus we come to a small conclusion that to obtain an optimal output performance, the electromechanical coupling factor δ should be set to an appropriate value. However, a direct calculation using the parameters introduced in the literature [1, 2] shows that the parameter δ is rather small for a typical piezoelectric cantilever energy harvester. For example, for a piezoelectric voltage constant $e_{31} = -5.35 \ C/m^2$, the value of e_p is -5.35×10^{-5} C, and the final value of δ is 0.028. According to the properties of commonly

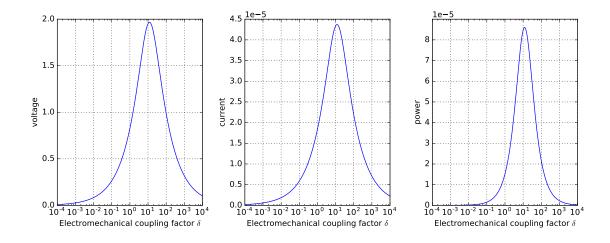


Figure 4: Voltage, current and power output for the piezoelectric cantilever energy harvester

used piezoelectric materials, the parameter e_{31} is always in the range of several or several tens C/m^2 reference to be inserted. That is to say, the final value of δ can be seen always in the order of 10^{-2} , which is a rather small value according to the diagram. Hence we could present an asymptotic analysis of the performance of the classical piezoelectric energy harvester. This is the subject of the following section.

5 Asymptotic analysis of the problem

Considering that the parameter δ is small, we expand the theoretical solution to the problem in terms of the small parameter δ using the following regular expansion:

$$\begin{cases}
A_{\delta} = A_0 + \delta A_1 + \delta^2 A_2 + \cdots, \\
B_{\delta} = B_0 + \delta B_1 + \delta^2 B_2 + \cdots, \\
C_{\delta} = C_0 + \delta C_1 + \delta^2 C_2 + \cdots, \\
D_{\delta} = D_0 + \delta D_1 + \delta^2 D_2 + \cdots.
\end{cases}$$
(30)

As a result, we obtain the following successive expansion problem: $O(\delta^0)$:

$$\begin{cases}
A_0 + C_0 = 1, \\
B_0 + D_0 = 0, \\
-A_0 \cos \sqrt{\sigma} - B_0 \sin \sqrt{\sigma} + C_0 \cosh \sqrt{\sigma} + D_0 \sinh \sqrt{\sigma} = 0, \\
A_0 \sin \sqrt{\sigma} - B_0 \cos \sqrt{\sigma} + C_0 \sinh \sqrt{\sigma} + D_0 \cosh \sqrt{\sigma} = 0.
\end{cases}$$
(31)

The solution is

$$\begin{cases}
A_{0} = \frac{1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma} - \sin\sqrt{\sigma}\sinh\sqrt{\sigma}}{2 + 2\cos\sqrt{\sigma}\cosh\sqrt{\sigma}}, \\
B_{0} = \frac{\cosh\sqrt{\sigma}\sin\sqrt{\sigma} + \cos\sqrt{\sigma}\sinh\sqrt{\sigma}}{2 + 2\cos\sqrt{\sigma}\cosh\sqrt{\sigma}}, \\
C_{0} = \frac{1 + \cos\sqrt{\sigma}\cosh\sqrt{\sigma}+\sin\sqrt{\sigma}\sinh\sqrt{\sigma}}{2 + 2\cos\sqrt{\sigma}\cosh\sqrt{\sigma}}, \\
D_{0} = -\frac{\cosh\sqrt{\sigma}\sin\sqrt{\sigma} + \cos\sqrt{\sigma}\sinh\sqrt{\sigma}}{2 + 2\cos\sqrt{\sigma}\cosh\sqrt{\sigma}}.
\end{cases}$$
(32)

Hence we have

$$-A_0 \sin \sqrt{\sigma} + B_0 \cos \sqrt{\sigma} + C_0 \sinh \sqrt{\sigma} + D_0 \cosh \sqrt{\sigma} = \frac{\sinh \sqrt{\sigma} - \sin \sqrt{\sigma}}{\cos \sqrt{\sigma} \cosh \sqrt{\sigma} + 1}$$
(33)

 $O(\delta^1)$:

$$\begin{cases}
A_1 + C_1 = 0, \\
B_1 + D_1 = 0, \\
(-A_1 \cos \sqrt{\sigma} - B_1 \sin \sqrt{\sigma} + C_1 \cosh \sqrt{\sigma} + D_1 \sinh \sqrt{\sigma}) + \\
\frac{j\beta\sqrt{\sigma}}{j\sigma\beta + 1} \left(-A_0 \sin \sqrt{\sigma} + B_0 \cos \sqrt{\sigma} + C_0 \sinh \sqrt{\sigma} + D_0 \cosh \sqrt{\sigma} \right) = 0, \\
A_1 \sin \sqrt{\sigma} - B_1 \cos \sqrt{\sigma} + C_1 \sinh \sqrt{\sigma} + D_1 \cosh \sqrt{\sigma} = 0.
\end{cases}$$
(34)

The solution is

$$\begin{cases}
A_1 = \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma} \left(\frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1} \right) \left(\frac{\cos\sqrt{\sigma} + \cosh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2} \right) \\
B_1 = \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma} \left(\frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1} \right) \left(\frac{-\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2} \right) \\
C_1 = \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma} \left(\frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1} \right) \left(-\frac{\cos\sqrt{\sigma} + \cosh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2} \right) \\
D_1 = \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma} \left(\frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1} \right) \left(\frac{-\sin\sqrt{\sigma} + \sinh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2} \right)
\end{cases} (35)$$

Then we have

$$-A_1 \sin \sqrt{\sigma} + B_1 \cos \sqrt{\sigma} + C_1 \sinh \sqrt{\sigma} + D_1 \cosh \sqrt{\sigma}$$

$$= \frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma} \left(\frac{\sin \sqrt{\sigma} - \sinh \sqrt{\sigma}}{\cos \sqrt{\sigma} \cosh \sqrt{\sigma} + 1} \right) \left(\frac{\cos \sqrt{\sigma} \sinh \sqrt{\sigma} + \sin \sqrt{\sigma} \cosh \sqrt{\sigma}}{\cos \sqrt{\sigma} \cosh \sqrt{\sigma} + 1} \right)$$
(36)

 $O(\delta^2)$:

$$\begin{cases} A_2 + C_2 = 0, \\ B_2 + D_2 = 0, \end{cases}$$

$$\left(-A_2 \cos \sqrt{\sigma} - B_2 \sin \sqrt{\sigma} + C_2 \cosh \sqrt{\sigma} + D_2 \sinh \sqrt{\sigma} \right) + \\ \frac{j\beta\sqrt{\sigma}}{j\sigma\beta + 1} \left(-A_1 \sin \sqrt{\sigma} + B_1 \cos \sqrt{\sigma} + C_1 \sinh \sqrt{\sigma} + D_1 \cosh \sqrt{\sigma} \right) = 0, \\ A_2 \sin \sqrt{\sigma} - B_2 \cos \sqrt{\sigma} + C_2 \sinh \sqrt{\sigma} + D_2 \cosh \sqrt{\sigma} = 0. \end{cases}$$
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The solution is

solution is
$$\begin{cases}
A_2 = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right)^2 \left(\frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(\frac{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(\frac{\cos\sqrt{\sigma} + \cosh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2}\right) \\
B_2 = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right)^2 \left(\frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(\frac{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(\frac{-\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2}\right) \\
C_2 = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right)^2 \left(\frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(\frac{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(-\frac{\cos\sqrt{\sigma} + \cosh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2}\right) \\
D_2 = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right)^2 \left(\frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(\frac{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(\frac{-\sin\sqrt{\sigma} + \sinh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2}\right)
\end{cases}$$
(38)

Indeed, we can continue to obtain the coefficients for higher order (>2) expansions, as shown in the appendices (To add some comments) using successive iteration method. Nevertheless, it suffices here to consider up to the second order expansion $u^{(0)}(z)$, $u^{(1)}(z)$, and $u^{(2)}(z)$, respectively:

$$\begin{cases}
 u^{(0)}(z) = u_0(z), \\
 u^{(1)}(z) = u_0(z) + \delta u_1(z), \\
 u^{(2)}(z) = u_0(z) + \delta u_1(z) + \delta^2 u_2(z),
\end{cases}$$
(39)

where the terms $u_0(z)$, $u_1(z)$, and $u_2(z)$ are defined as

$$\begin{cases}
 u_0(z) = A_0 \cos \sqrt{\sigma}z + B_0 \sin \sqrt{\sigma}z + C_0 \cosh \sqrt{\sigma}z + D_0 \sinh \sqrt{\sigma}z - 1, \\
 u_1(z) = A_1 \cos \sqrt{\sigma}z + B_1 \sin \sqrt{\sigma}z + C_1 \cosh \sqrt{\sigma}z + D_1 \sinh \sqrt{\sigma}z, \\
 u_2(z) = A_2 \cos \sqrt{\sigma}z + B_2 \sin \sqrt{\sigma}z + C_2 \cosh \sqrt{\sigma}z + D_2 \sinh \sqrt{\sigma}z.
\end{cases} (40)$$

For different values of δ and σ (Here in this simulation, the value of σ is changed through the variance of base excitation frequency f_b), the asymptotic approximations of the dimensionless relative beam displacement function $u(z;\delta)$ up to the second order $u^{(0)}(z)$, $u^{(1)}(z)$, and $u^{(2)}(z)$ are calculated and compared to the closed solution $u(z;\delta)$ itself. The results are shown in Figure 5, 6, 7, 8, 9, 10, 11.

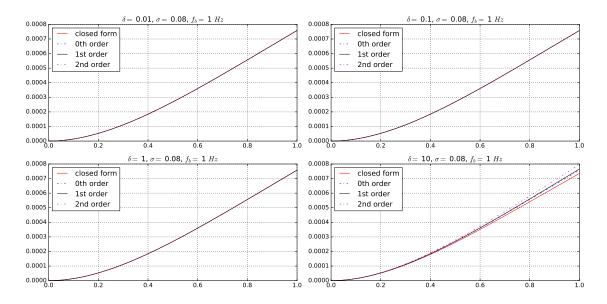


Figure 5: Comparison for the different orders of asymptotic expansion for the dimensionless relative displacement function $u_{\delta}(z)$.

Notice from Figure 1 that the first mode resonant frequency for the device is around 45 Hz. It is seen from these results that a smaller value of δ results in a better approximation, whatever the value of f_b . This is consistent with the philosophy behind asymptotic expansion. Besides, for the frequency away from the resonant frequency, the approximation results are relatively accurate in the range of $\delta \leq 0.1$ depending on the value of f_b . While when the base exciation frequency f_b is close to a resonance, for exmaple $f_b = 45 Hz$, the approximation results are not accurate even at the value of $\delta = 0.01$. That is to say, the asymptotic expansion in terms of δ is not uniform with respect to parameter σ . Especially, the existence of resonance actually restrict the behavior of the asymptotic expansion. Around the resonance the expansion will show low accuracy, while away from the resonance, the expansion accuracy is easily retained. Nonetheless, for commonly used piezoelectric materials and energy harvesting device configuration, the value of δ is usually in the range of 10^{-2} . Hence, it is generally validated to use the asymptotic expansion method to approximate the dimensionless displacement function $u(z;\delta)$. Furtherly, in view of the approximating performances of the asymptotic expansions to different orders, it suffices to keep only the 0th order terms. Then, we have that

$$u(z;\delta) \approx u^{(0)}(z) = A_0 \cos \sqrt{\sigma}z + B_0 \sin \sqrt{\sigma}z + C_0 \cosh \sqrt{\sigma}z + D_0 \sinh \sqrt{\sigma}z - 1.$$
 (41)

This is exactly the displacement function of a pure elastic cantilever beam. It means that for most piezoelectric energy harvesting devices, due to the fact that the electromechanical coupling factor is relatively small, the displacement function is not much affected. In this way, we have indeed uncoupled the electrical part and elastic part of a piezoelectric energy harvesting device. It should be noted that the approximation is not valid near the resonant points.

Theoretically, the first order derivative of the dimensionless relative displacement funtion $u(z;\delta)$ is

$$u'(z;\delta) = \sigma^{1/2} \left(-A_{\delta} \sin \sqrt{\sigma} z + B_{\delta} \cos \sqrt{\sigma} z + C_{\delta} \sinh \sqrt{\sigma} z + D_{\delta} \cosh \sqrt{\sigma} z \right). \tag{42}$$

$$\chi_p = u_1'(1) = \frac{\sqrt{\sigma} \left(\sinh \sqrt{\sigma} - \sin \sqrt{\sigma}\right)}{1 + \cos \sqrt{\sigma} \cosh \sqrt{\sigma} + \frac{j\beta\sqrt{\sigma}}{1 + j\beta\sigma} \delta \left(\cos \sqrt{\sigma} \sinh \sqrt{\sigma} + \sin \sqrt{\sigma} \cosh \sqrt{\sigma}\right)}.$$
 (43)

The value of this function at the free end (z=1) is just the output index χ_p according to equation (29). To see the influences of δ and f_b , therefore σ , upon χ_p , we firstly calculate the values of χ_p

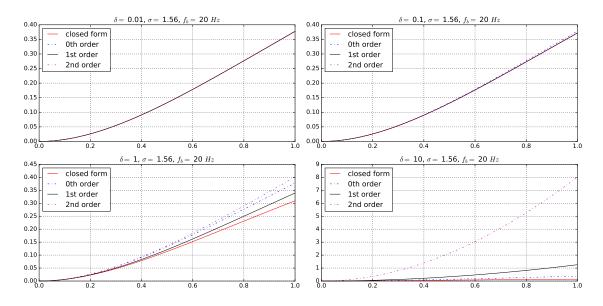


Figure 6: Comparison for the different orders of asymptotic expansion for the dimensionless relative displacement function $u_{\delta}(z)$.

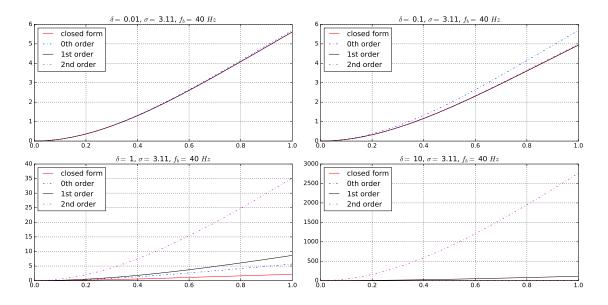


Figure 7: Comparison for the different orders of asymptotic expansion for the dimensionless relative displacement function $u_{\delta}(z)$.

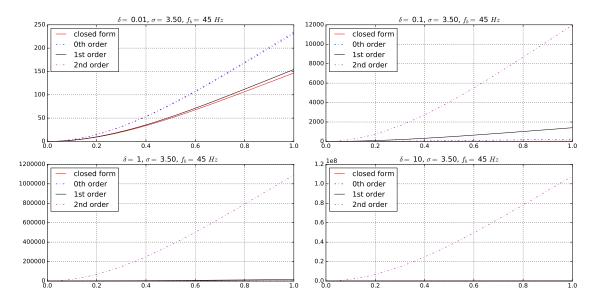


Figure 8: Comparison for the different orders of asymptotic expansion for the dimensionless relative displacement function $u_{\delta}(z)$.

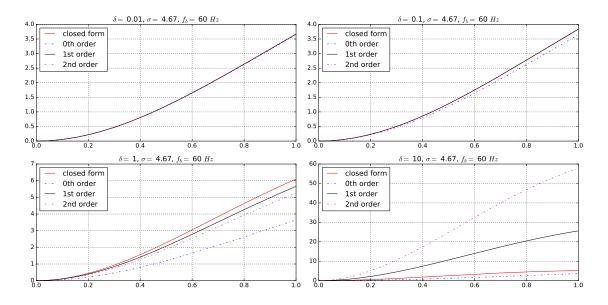


Figure 9: Comparison for the different orders of asymptotic expansion for the dimensionless relative displacement function $u_{\delta}(z)$.

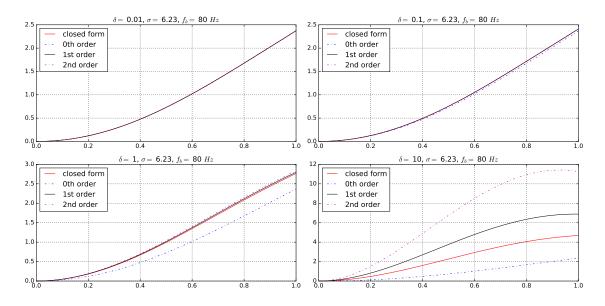


Figure 10: Comparison for the different orders of asymptotic expansion for the dimensionless relative displacement function $u_{\delta}(z)$.

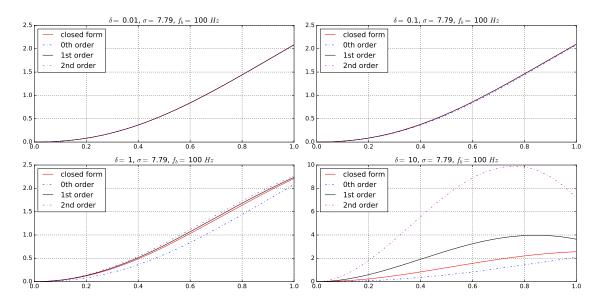


Figure 11: Comparison for the different orders of asymptotic expansion for the dimensionless relative displacement function $u_{\delta}(z)$.

for different values of δ by fixing the value of f_b to be some discrete values. The results are shown in Figure 12.

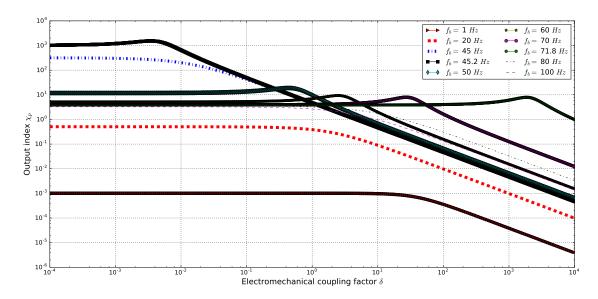


Figure 12: Output index χ_p as a function of electromechanical coupling factor δ at different values of base excitation frequency f_b .

It is seen that the dependence of χ_p upon δ shows two different modes in the frequency range of 1-100~Hz. For the frequencies of $45.2~Hz \le f_b \le 71.8~Hz$, a peak corresponding to a critical value of δ_p is present in the considered range of δ . When δ is smaller than δ_p , the output index χ_p increases along with δ , and when δ is larger than δ_p , the output index χ_p decreases with the increase of δ . On the other hand, for the frequency range of $f_b \le 45~Hz$ or $f_b \ge 80~Hz$, the output index χ_p shows a monotonical decrease with respect to the increase δ .

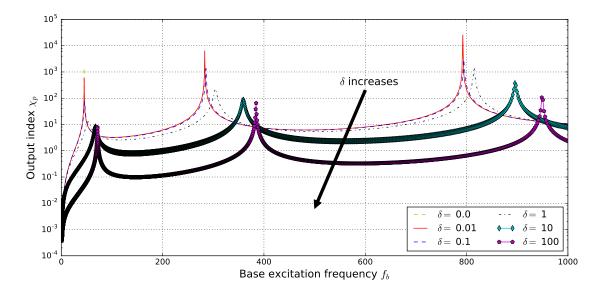


Figure 13: Output index χ_p as a function of base excitation frequency f_b at different values of electromechanical coupling factor δ .

Alternatively, by fixing the values of δ to a discrete set of numbers, we calculate the values of χ_p in relation to different values of f_b . This is very similar to a frequency response of the output index as a function of σ . The results are shown in Figure 13. It is clearly shown that with the increase of electromechanical coupling factor δ , the resonant frequencies to the system increase with the increase of δ . This is clearly shown in the shift to the right of the frequency response curve.

Besides, we add in this figure the case where $\delta=0.0$ as a reference. Simple comparisons show that the discrepency between the frequency response curves related to the case of $\delta=0$, $\delta=0.01$, and $\delta=0.1$ is small. A direct conclusion is that for relatively small values of electromechanical coupling factor δ , the output index χ_p can be approximated by

$$\chi_p \approx \frac{\sqrt{\sigma} \left(\sinh \sqrt{\sigma} - \sin \sqrt{\sigma}\right)}{1 + \cos \sqrt{\sigma} \cosh \sqrt{\sigma}}.$$
(44)

As a result, the output performance measures \tilde{V}_p , \tilde{I}_p , and \tilde{P}_p can be approximated by

$$\begin{cases}
\tilde{V}_{p} = -\frac{j\sigma\beta}{j\sigma\beta + 1} \left(\frac{\eta_{b}}{l_{p}}\right) \left(\frac{e_{p}}{C_{p}}\right) \chi_{p}, \\
= -\frac{j\sigma\beta}{j\sigma\beta + 1} \left(\frac{\eta_{b}}{l_{p}}\right) \left(\frac{e_{p}}{C_{p}}\right) \frac{\sqrt{\sigma} \left(\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}\right)}{1 + \cos\sqrt{\sigma} \cosh\sqrt{\sigma}}, \\
\tilde{I}_{p} = \tilde{V}_{p}/R_{l} = -\frac{j\sigma\beta}{j\sigma\beta + 1} \left(\frac{\eta_{b}}{l_{p}}\right) \left(\frac{e_{p}}{C_{p}R_{l}}\right) \chi_{p}, \\
= -\frac{j\sigma\beta}{j\sigma\beta + 1} \left(\frac{\eta_{b}}{l_{p}}\right) \left(\frac{e_{p}}{C_{p}R_{l}}\right) \frac{\sqrt{\sigma} \left(\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}\right)}{1 + \cos\sqrt{\sigma} \cosh\sqrt{\sigma}}, \\
\tilde{P}_{p} = \tilde{V}_{p}^{2}/R_{l} = \left(\frac{\eta_{b}}{l_{p}}\right)^{2} \left(\frac{e_{p}}{C_{p}}\right) \left(\frac{e_{p}}{C_{p}R_{l}}\right) \left(\frac{j\sigma\beta}{j\sigma\beta + 1}\right)^{2} \chi_{p}^{2}, \\
= \left(\frac{\eta_{b}}{l_{p}}\right)^{2} \left(\frac{e_{p}}{C_{p}}\right) \left(\frac{e_{p}}{C_{p}R_{l}}\right) \left(\frac{j\sigma\beta}{j\sigma\beta + 1}\right)^{2} \left(\frac{\sqrt{\sigma} \left(\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}\right)}{1 + \cos\sqrt{\sigma} \cosh\sqrt{\sigma}}\right)^{2}.
\end{cases}$$

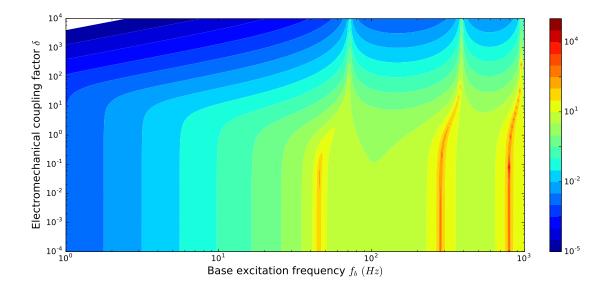


Figure 14: Output index χ_p as a function of base excitation frequency f_b and electromechanical coupling factor δ .

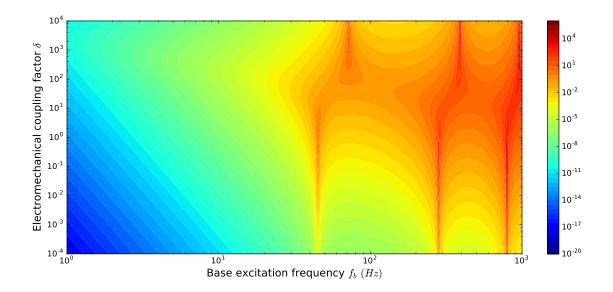


Figure 15: Output voltage \tilde{V}_p as a function of base excitation frequency f_b and electromechanical coupling factor δ .

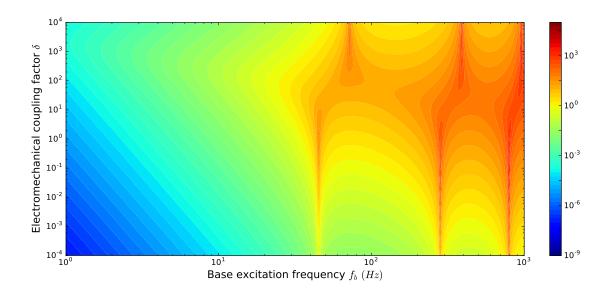


Figure 16: Output voltage \tilde{P}_p as a function of base excitation frequency f_b and electromechanical coupling factor δ .

Conclusion 6

Appendices

The asymptotic expansion of equation (23) can be found using an iterative method. In fact, for higher order expansions $(k \geq 1)$, we have the following iterative relation:

$$A_{k+1} + C_{k+1} = 0,$$

$$B_{k+1} + D_{k+1} = 0,$$

$$\left(-A_{k+1}\cos\sqrt{\sigma} - B_{k+1}\sin\sqrt{\sigma} + C_{k+1}\cosh\sqrt{\sigma} + D_{k+1}\sinh\sqrt{\sigma}\right) +$$

$$\frac{j\beta\sqrt{\sigma}}{j\sigma\beta + 1}\left(-A_k\sin\sqrt{\sigma} + B_k\cos\sqrt{\sigma} + C_k\sinh\sqrt{\sigma} + D_k\cosh\sqrt{\sigma}\right) = 0,$$

$$A_{k+1}\sin\sqrt{\sigma} - B_{k+1}\cos\sqrt{\sigma} + C_{k+1}\sinh\sqrt{\sigma} + D_{k+1}\cosh\sqrt{\sigma} = 0,$$

$$(46)$$

whose solution is expressed by

$$\begin{cases}
A_{k+1} = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right) \left(\frac{\cos\sqrt{\sigma} + \cosh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2}\right) (Q_k), \\
B_{k+1} = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right) \left(\frac{-\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2}\right) (Q_k), \\
C_{k+1} = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right) \left(-\frac{\cos\sqrt{\sigma} + \cosh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2}\right) (Q_k), \\
D_{k+1} = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right) \left(\frac{-\sin\sqrt{\sigma} + \sinh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 2}\right) (Q_k),
\end{cases}$$

$$(47)$$

in which

$$Q_k = -A_k \sin \sqrt{\sigma} + B_k \cos \sqrt{\sigma} + C_k \sinh \sqrt{\sigma} + D_k \cosh \sqrt{\sigma}.$$
(48)

In terms of
$$Q_k$$
 $(k \ge 0)$, we have the following iterative relation
$$Q_{k+1} = -\left(\frac{\sin\sqrt{\sigma}\cosh\sqrt{\sigma} + \cos\sqrt{\sigma}\sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right) Q_k,\tag{49}$$

and the initial two values Q_0 and Q_1 :

$$\begin{cases}
Q_0 = \frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}, \\
Q_1 = \frac{j\beta\sqrt{\sigma}}{1 + j\beta\sigma} \left(\frac{\sin\sqrt{\sigma} - \sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right) \left(\frac{\cos\sqrt{\sigma}\sinh\sqrt{\sigma} + \sin\sqrt{\sigma}\cosh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1}\right).
\end{cases} (50)$$

Hence it is shown that for $k \geq 0$,

$$Q_k = \left[-\left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma} \right) \left(\frac{\sin\sqrt{\sigma}\cosh\sqrt{\sigma} + \cos\sqrt{\sigma}\sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1} \right) \right]^k \left(\frac{\sinh\sqrt{\sigma} - \sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma} + 1} \right). \tag{51}$$

As a result, we obtain that for

$$\begin{cases} A_k = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right)^k \left(\frac{-\sin\sqrt{\sigma}\cosh\sqrt{\sigma}-\cos\sqrt{\sigma}\sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+1}\right)^{k-1} \left(\frac{\sinh\sqrt{\sigma}-\sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+1}\right) \left(\frac{\cos\sqrt{\sigma}+\cosh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+2}\right), \\ B_k = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right)^k \left(\frac{-\sin\sqrt{\sigma}\cosh\sqrt{\sigma}-\cos\sqrt{\sigma}\sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+1}\right)^{k-1} \left(\frac{\sinh\sqrt{\sigma}-\sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+1}\right) \left(\frac{-\sinh\sqrt{\sigma}+\sin\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+2}\right), \\ C_k = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right)^k \left(\frac{-\sin\sqrt{\sigma}\cosh\sqrt{\sigma}-\cos\sqrt{\sigma}\sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+1}\right)^{k-1} \left(\frac{\sinh\sqrt{\sigma}-\sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+1}\right) \left(\frac{-\cos\sqrt{\sigma}-\cosh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+2}\right), \\ D_k = \left(\frac{j\beta\sqrt{\sigma}}{1+j\beta\sigma}\right)^k \left(\frac{-\sin\sqrt{\sigma}\cosh\sqrt{\sigma}-\cos\sqrt{\sigma}\sinh\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+1}\right)^{k-1} \left(\frac{\sinh\sqrt{\sigma}-\sin\sqrt{\sigma}}{\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+1}\right) \left(\frac{-\sin\sqrt{\sigma}+\sinh\sqrt{\sigma}}{2\cos\sqrt{\sigma}\cosh\sqrt{\sigma}+2}\right). \end{cases}$$

Acknowledgements

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[1] Erturk A, Inman DJ. A distributed parameter electromechanical model for cantilevered piezoelectric energy harvesters. Journal of vibration and acoustics. 2008;130(4):041002.

energy	harvesting	from	base excitations.	Smart	materials	and st	ructures.	2009;18(2)):025009.	

[2] Erturk A, Inman DJ. An experimentally validated bimorph cantilever model for piezoelectric