Computer Graphics I

Lecture 21: Fluid simulation

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What is a fluid?

• Definition

 A substance that continually deforms (flows) under an applied stress



Fluid simulation

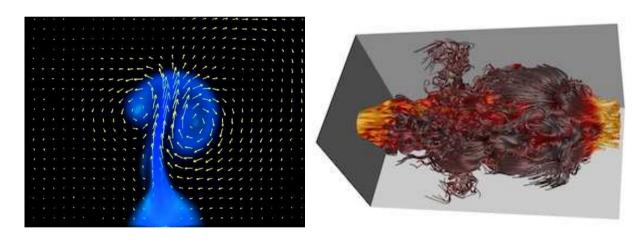
Simulate the behavior of fluid flows

- Modeling the fluid behavior
- Solve the fluid equations numerically
- Rendering the simulation result



Physical quantities

- Velocity field u
 - A spatially and temporally varying vector field of velocity

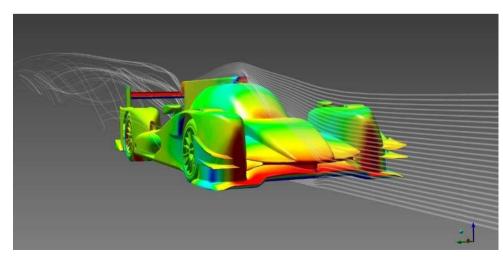




Physical quantities

• Likewise

- Pressure field p
 - A scalar field of pressure
- Density field ρ
 - A scalar field of density
- Temperature field T
 - A scalar field of temperature



Pressure field on a car surface

Physical quantities

Viscosity ν

- A measure of its resistance to gradual deformation
- Small viscosity implies easier (large) deformation



Divergence

 The divergence represents the volume density of the outward flux of a vector field from an infinitesimal volume

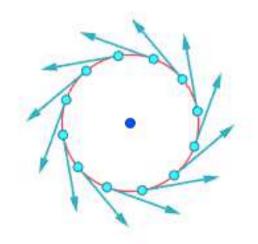
$$\left. \operatorname{div} \mathbf{F}
ight|_p = \lim_{V o \{p\}} \iint_{S(V)} rac{\mathbf{F} \cdot \hat{\mathbf{n}}}{|V|} \, dS$$

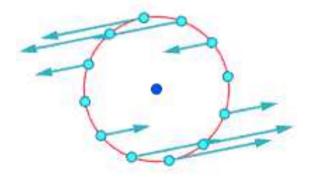
$$\operatorname{div} \mathbf{F} =
abla \cdot \mathbf{F} = \left(\frac{\partial}{\partial x}, \frac{\partial}{\partial y}, \frac{\partial}{\partial z} \right) \cdot (U, V, W) = \frac{\partial U}{\partial x} + \frac{\partial V}{\partial y} + \frac{\partial W}{\partial z}.$$

Vorticity

Describes the local spinning motion

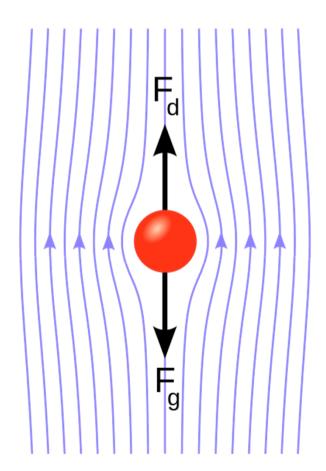
$$egin{aligned} ec{\omega} &=
abla imes ec{v} = \left(rac{\partial}{\partial x}, rac{\partial}{\partial y}, rac{\partial}{\partial z}
ight) imes (v_x, v_y, v_z) \ &= \left(rac{\partial v_z}{\partial y} - rac{\partial v_y}{\partial z}, rac{\partial v_x}{\partial z} - rac{\partial v_z}{\partial x}, rac{\partial v_y}{\partial x} - rac{\partial v_x}{\partial y}
ight) \end{aligned}$$

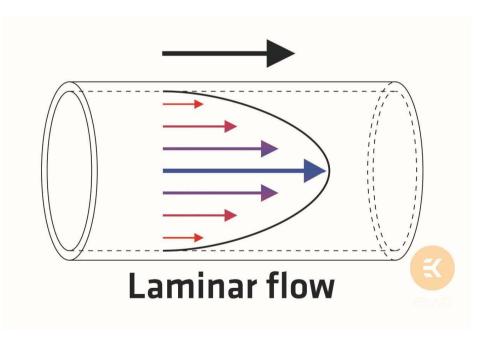




Laminar flow

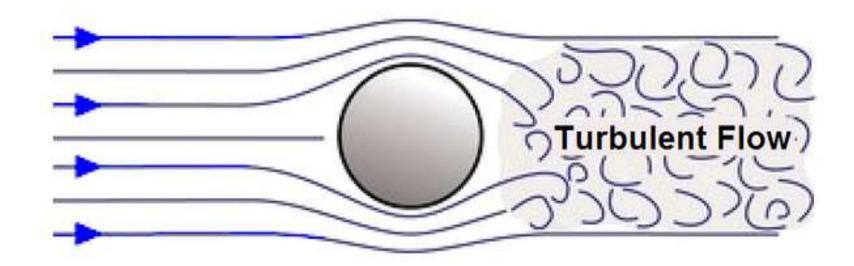
- When a fluid flows in parallel layers
 - No disruption between the layers





Turbulence flow

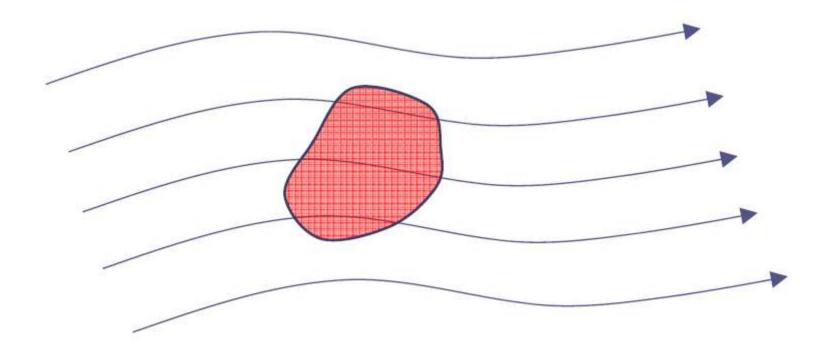
• A flow in chaotic behavior



1. Modeling for Fluids

Control volume

• An enclosed volume with arbitrary shape

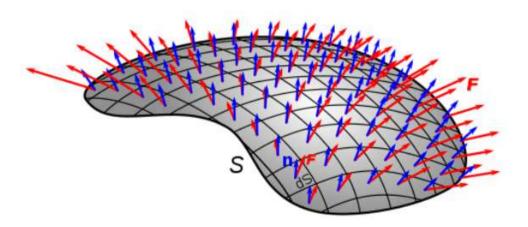


Flux

Any effect that passes through a surface or

substance

$$\varphi = \oint_{S} \mathbf{Q} \cdot d\mathbf{s}$$



Fluid dynamics

- Conservation laws
 - Conservation of mass
 - The rate of change of mass = mass flux rate
 - Conservation of momentum
 - The rate of change of momentum = internal + external force

Conservation of mass

- Continuity equation
 - Integral form

$$\frac{\partial}{\partial t} \oint_{V} \rho dV = -\oint_{S} \rho \mathbf{u} \cdot d\mathbf{s}$$

Differential form

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$$

Conservation of momentum

Momentum equation

- Apply Newton's second law
- Integral form

$$\frac{\partial}{\partial t} \oint_{V} \rho \mathbf{u} dV + \oint_{s} (\rho \mathbf{u} \cdot d\mathbf{s}) \mathbf{u} = -\oint_{s} p d\mathbf{s} + \oint_{s} \mathbf{\tau}_{shear} \cdot d\mathbf{s} + \oint_{V} \rho \mathbf{g} dV$$

Differential form

$$\frac{\partial \rho \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla(\rho \mathbf{u}) = -\nabla p + \nabla \cdot \mathbf{\tau}_{shear} + \rho \mathbf{g}$$

Integral form

$$\frac{\partial}{\partial t} \oint_{V} \rho dV = -\oint_{S} \rho \mathbf{u} \cdot d\mathbf{s}$$

$$\frac{\partial}{\partial t} \oint_{V} \rho \mathbf{u} dV + \oint_{S} (\rho \mathbf{u} \cdot d\mathbf{s}) \mathbf{u} = -\oint_{S} p d\mathbf{s} + \oint_{S} \mathbf{\tau}_{shear} \cdot d\mathbf{s} + \oint_{V} \rho \mathbf{g} dV$$

Differential form

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$$

$$\frac{\partial \rho \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla(\rho \mathbf{u}) = -\nabla p + \nabla \cdot \mathbf{\tau}_{shear} + \rho \mathbf{g}$$

Incompressible fluids

- The volume and density do not change
 - Differential form (constant density)

$$\nabla \cdot \mathbf{u} = 0$$

$$\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} = -\frac{1}{\rho} \nabla p + \frac{1}{\rho} \nabla \cdot \boldsymbol{\tau}_{shear} + \mathbf{g}$$

Newtonian fluids

Linear shear-stress relation

$$\boldsymbol{\tau}_{shear} = -\boldsymbol{\nu}' \, \boldsymbol{\rho} (\nabla \cdot \mathbf{u}) \mathbf{I} + 2 \boldsymbol{\rho} \boldsymbol{\nu} \mathbf{S}$$

Strain rate (deformation rate)

$$\mathbf{S} = \frac{1}{2} \left(\nabla \mathbf{u} + (\nabla \mathbf{u})^T \right)$$

Incompressible Navier-Stokes equations

Apply Newtonion fluid relation in incompressible governing equations

$$\nabla \cdot \mathbf{u} = 0$$

$$\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} = -\frac{1}{\rho} \nabla p + \nu \nabla^2 \mathbf{u} + \mathbf{g}$$

Incompressible Navier-Stokes equations

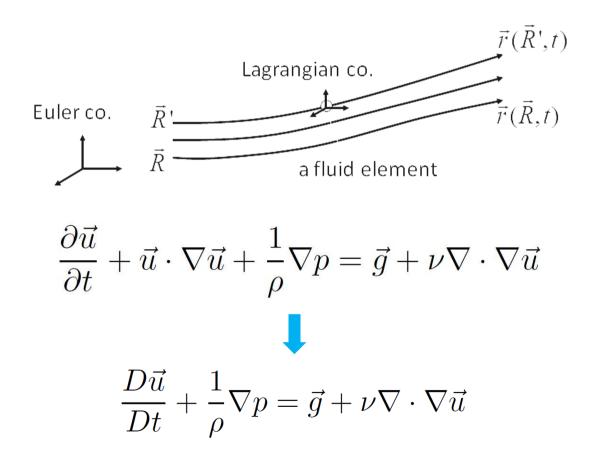
- Fluid equations in computer graphics
 - Navier-Stokes equations in isothermal case

$$\frac{\partial \vec{u}}{\partial t} + \vec{u} \cdot \nabla \vec{u} + \frac{1}{\rho} \nabla p = \vec{g} + \nu \nabla \cdot \nabla \vec{u}$$

$$\nabla \cdot \vec{u} = 0$$

Incompressibility constraint

- Lagrangian view
 - Coordinate is moving with fluid



Pressure equation

Taking divergence of the momentum equation

$$\nabla \cdot \frac{\partial \vec{u}}{\partial t} + \nabla \cdot (\vec{u} \cdot \nabla \vec{u}) + \nabla \cdot \frac{1}{\rho} \nabla p = \nabla \cdot (\vec{g} + \nu \nabla \cdot \nabla \vec{u})$$

Remove zero terms

$$\nabla \cdot \vec{u} = 0 \longrightarrow \nabla \cdot \frac{1}{\rho} \nabla p = \nabla \cdot (-\vec{u} \cdot \nabla \vec{u} + \vec{g} + \nu \nabla \cdot \nabla \vec{u})$$

Ideal inviscid flows

- No viscosity or very small viscosity
- Drop the viscosity term

$$\frac{D\vec{u}}{Dt} + \frac{1}{\rho}\nabla p = \vec{g}$$
$$\nabla \cdot \vec{u} = 0$$



Boundary conditions

- Solid wall boundary
 - The flow cannot penetrate
 - Slip boundary

$$\vec{u} \cdot \hat{n} = \vec{u}_{\text{solid}} \cdot \hat{n}$$

No-slip (Neumann) boundary

$$ec{u}=ec{u}_{ ext{Solid}}$$
 large moving plate velocity at wall equals velocity of moving plate velocity at wall is zero

2. Unconditionally stable semi-Lagrangian method

Finite difference method

- Derivative approximation
 - First derivative
 - Taylor expansion

$$u(x + \Delta x) = u(x) + \Delta x \frac{\partial u(x)}{\partial x} + \frac{(\Delta x)^2}{2} \frac{\partial^2 u(x)}{\partial x^2} + \frac{(\Delta x)^3}{3!} \frac{\partial^3 u(x)}{\partial x^3} + \cdots$$

First order approximation

$$\frac{u(x + \Delta x) - u(x)}{\Delta x} = \frac{\partial u(x)}{\partial x} + \frac{\Delta x}{2} \frac{\partial^2 u(x)}{\partial x^2} + \dots = \frac{\partial u(x)}{\partial x} + O(\Delta x)$$

Finite difference method

Derivative approximation

- First derivative
 - Second order approximation

$$u_{i+1} = u_i + \Delta x \left(\frac{\partial u}{\partial x}\right)_i + \frac{\Delta x^2}{2} \left(\frac{\partial^2 u}{\partial x^2}\right)_i + \frac{\Delta x^3}{3!} \left(\frac{\partial^3 u}{\partial x^3}\right)_i + \frac{\Delta x^4}{4!} \left(\frac{\partial^4 u}{\partial x^4}\right)_i + \cdots$$

$$u_{i-1} = u_i - \Delta x \left(\frac{\partial u}{\partial x}\right)_i + \frac{\Delta x^2}{2} \left(\frac{\partial^2 u}{\partial x^2}\right)_i - \frac{\Delta x^3}{3!} \left(\frac{\partial^3 u}{\partial x^3}\right)_i + \frac{\Delta x^4}{4!} \left(\frac{\partial^4 u}{\partial x^4}\right)_i + \cdots$$



$$\left(\frac{\partial u}{\partial x}\right)_i = \frac{u_{i+1} - u_{i-1}}{2\Delta x} + \mathcal{O}(\Delta x^2)$$

Splitting

- How to solve the fluid equations?
 - Split the complex equations into simpler ones
- Splitting in general PDE

$$\frac{dq}{dt} = f(q) + g(q)$$

$$\tilde{q} = q^n + \Delta t f(q^n)$$

$$q^{n+1} = \tilde{q} + \Delta t g(\tilde{q})$$

Splitting

- Splitting in general PDE
 - Is splitting with the same order as original? Yes!

$$q^{n+1} = (q^n + \Delta t f(q^n)) + \Delta t g(q^n + \Delta t f(q^n))$$

$$= q^n + \Delta t f(q^n) + \Delta t (g(q^n) + O(\Delta t))$$

$$= q^n + \Delta t (f(q^n) + g(q^n)) + O(\Delta t^2)$$

$$= q^n + \frac{dq}{dt} \Delta t + O(\Delta t^2)$$

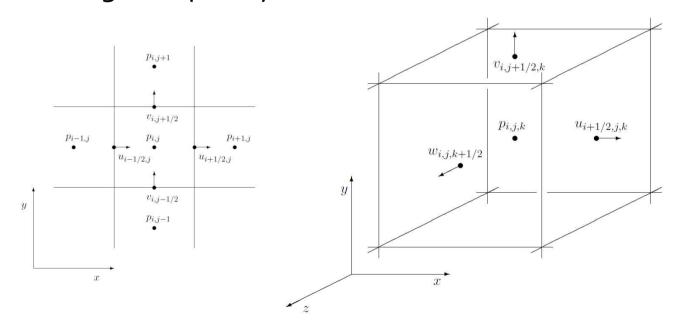
Splitting for Navier-Stokes equations

$$\mathbf{w}_0(\mathbf{x}) \overset{\text{add force}}{\longrightarrow} \mathbf{w}_1(\mathbf{x}) \overset{\text{advect}}{\longrightarrow} \mathbf{w}_2(\mathbf{x}) \overset{\text{diffuse}}{\longrightarrow} \mathbf{w}_3(\mathbf{x}) \overset{\text{project}}{\longrightarrow} \mathbf{w}_4(\mathbf{x})$$

MAC grid

Staggered grid

- Staggered arrangement of velocity and pressure
- Face center: velocity samples
- Cell center: pressure samples
- Avoid high frequency artifacts



Advection

- What is an advection?
 - Transport of a substance

$$Dq/Dt = 0$$

- Numerical solver
 - The simple discretization: explicit Euler

$$\frac{\partial q}{\partial t} + u \frac{\partial q}{\partial x} = 0 \qquad \longrightarrow \qquad \frac{q_i^{n+1} - q_i^n}{\Delta t} + u_i^n \frac{q_{i+1}^n - q_{i-1}^n}{2\Delta x} = 0$$

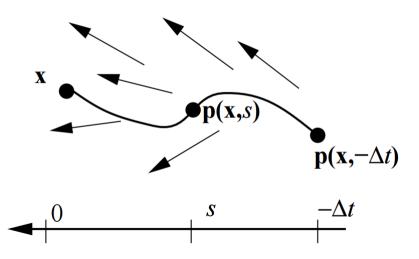


Unconditionally unstable! \longrightarrow $q_i^{n+1} = q_i^n - \Delta t u_i^n \frac{q_{i+1}^n - q_{i-1}^n}{2\Delta x}$

Semi-Lagrangian advection

- How to make stable solution?
 - Implicit formulation?
 - Difficult to solve; hard for non-linear advection
- Unconditionally stable yet simple solution?
 - Semi-Lagrangian scheme
 - Jos Stam, Stable Fluids, SIGGRAPH 1999

$$\frac{d\vec{x}}{dt} = \vec{u}$$



Unconditionally stable simulation

- Helmholtz-Hodge Decomposition
 - Any vector field can be uniquely decomposed into
 - Divergence free field
 - Divergent field

$$\mathbf{w} = \mathbf{u} + \nabla q \qquad \nabla \cdot \mathbf{u} = 0$$

- How can this be applied to fluid equations?
 - Define an operator P: project any vector field w onto its divergence free part u

$$\nabla \cdot \mathbf{w} = \nabla^2 q$$
 \longrightarrow $\mathbf{u} = \mathbf{P} \mathbf{w} = \mathbf{w} - \nabla q$

Unconditionally stable simulation

- Apply to Navier-Stokes equation
 - Apply to both sides of momentum equation

$$\frac{\partial \mathbf{u}}{\partial t} = -(\mathbf{u} \cdot \nabla)\mathbf{u} - \frac{1}{\rho}\nabla p + \nu\nabla^2\mathbf{u} + \mathbf{f}$$

$$\mathbf{P}\mathbf{u} = \mathbf{u} \quad \mathbf{P}\nabla p = 0 \qquad \mathbf{P}$$

$$\frac{\partial \mathbf{u}}{\partial t} = \mathbf{P}\left(-(\mathbf{u} \cdot \nabla)\mathbf{u} + \nu\nabla^2\mathbf{u} + \mathbf{f}\right)$$

$$\mathbf{w}_0(\mathbf{x}) \xrightarrow{\text{add force}} \mathbf{w}_1(\mathbf{x}) \xrightarrow{\text{advect}} \mathbf{w}_2(\mathbf{x}) \xrightarrow{\text{diffuse}} \mathbf{w}_3(\mathbf{x}) \xrightarrow{\text{project}} \mathbf{w}_4(\mathbf{x})$$

Unconditionally stable simulation

$$\frac{\partial \mathbf{u}}{\partial t} = \mathbf{P} \left(-(\mathbf{u} \cdot \nabla)\mathbf{u} + \nu \nabla^2 \mathbf{u} + \mathbf{f} \right)$$

- Add force $\mathbf{w}_1(\mathbf{x}) = \mathbf{w}_0(\mathbf{x}) + \Delta t \ \mathbf{f}(\mathbf{x}, t)$
- Advect $\mathbf{w}_2(\mathbf{x}) = \mathbf{w}_1(\mathbf{p}(\mathbf{x}, -\Delta t))$
- Diffuse $\left(\mathbf{I} \nu \Delta t \nabla^2\right) \mathbf{w}_3(\mathbf{x}) = \mathbf{w}_2(\mathbf{x})$
- Project $\nabla^2 q = \nabla \cdot \mathbf{w}_3$ $\mathbf{w}_4 = \mathbf{w}_3 \nabla q$

Vorticity confinement

- Compensate for numerical diffusion
 - Add a "vorticity confinement" force
 - Disturb the flow from current vorticity
 - Definition of vorticity
 - Find vortices $\vec{\omega} = \nabla \times \vec{u}$
 - The local axes of vorticity $\vec{N} = \frac{\nabla |\vec{\omega}|}{\|\nabla |\vec{\omega}|\|}$
 - Construct disturbing force

$$f_{\rm conf} = \epsilon \Delta x (\vec{N} \times \vec{\omega})$$

Vorticity confinement

Implementation

Computation of vorticity

$$\vec{\omega}_{i,j,k} = \left(\frac{w_{i,j+1,k} - w_{i,j-1,k}}{2\Delta x} - \frac{v_{i,j,k+1} - v_{i,j,k-1}}{2\Delta x}, \frac{u_{i,j,k+1} - u_{i,j,k-1}}{2\Delta x} - \frac{w_{i+1,j,k} - w_{i-1,j,k}}{2\Delta x}, \frac{v_{i+1,j,k} - v_{i-1,j,k}}{2\Delta x} - \frac{u_{i,j+1,k} - u_{i,j-1,k}}{2\Delta x}\right)$$

Computation of vorticity gradient

$$\nabla |\vec{\omega}|_{i,j,k} = \left(\frac{|\vec{\omega}|_{i+1,j,k} - |\vec{\omega}|_{i-1,j,k}}{2\Delta x}, \quad \frac{|\vec{\omega}|_{i,j+1,k} - |\vec{\omega}|_{i,j-1,k}}{2\Delta x}, \quad \frac{|\vec{\omega}|_{i,j,k+1} - |\vec{\omega}|_{i,j,k-1}}{2\Delta x}\right)$$

Normalize this to get N

$$\vec{N}_{i,j,k} = \frac{\nabla |\vec{\omega}|_{i,j,k}}{\|\nabla |\vec{\omega}|_{i,j,k}\| + 10^{-20}}$$

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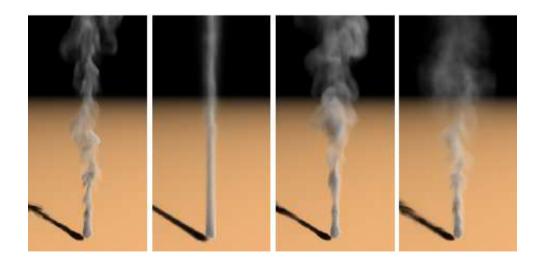
Smoke simulation

Evolve additional quantities

Temperature T and smoke concentration s

$$\frac{DT}{Dt} = 0 \qquad \frac{Ds}{Dt} = 0$$

- Buoyancy force $f_{\text{buoy}} = (0, -\alpha s + \beta (T - T_{\text{amb}}), 0)$



An Advection-Reflection Solver for Detail-Preserving Fluid Simulation

Jonas Zehnder Université de Montréal Rahul Narain Indian Institute of Technology Delhi University of Minnesota Bernhard Thomaszewski Université de Montréal

4. Liquid Simulation

Liquid simulation

• Characteristics of liquids

- Interface dynamics
- Tracking the motion of interface

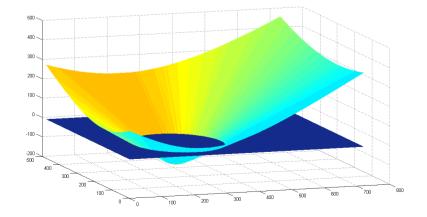


Liquid simulation

- Implicit surface & level set
 - Implicit surface
 - A surface in Euclidean space defined by an equation

$$F(x,y,z)=0.$$

- Why? Easy for complex surfaces
- Level set surface



Liquid simulation

Interface representation

- Implicit surface
- liquid volume as one side of an isocontour of an implicit function

$$\phi \leq 0$$

- Interface
 - Isosurface (isocontuour)

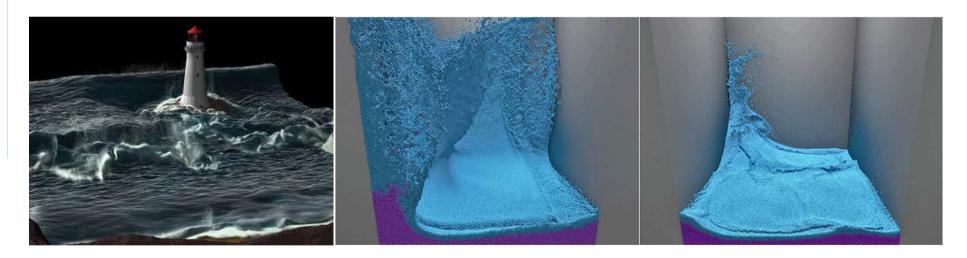
$$\phi = 0$$

- Interface advection
 - Level set equation

$$\phi_t + \vec{u} \cdot \nabla \phi = 0$$

Problem with mesh-based method

- How to simulate complex free surface?
 - Complex free boundary condition
 - Water splashes

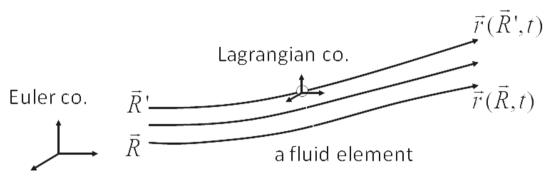


Particle methods are preferred

5. Smoothed Particle Hydrodynamics

Lagrangian solver for fluids

Coordinate is moving with fluid



$$\frac{\partial \vec{u}}{\partial t} + \vec{u} \cdot \nabla \vec{u} + \frac{1}{\rho} \nabla p = \vec{g} + \nu \nabla \cdot \nabla \vec{u}$$

$$\frac{D\vec{u}}{Dt} + \frac{1}{\rho} \nabla p = \vec{g} + \nu \nabla \cdot \nabla \vec{u}$$



Function approximation with SPH

- Problem setting
 - Reconstructing an (unknown) function f from a set of irregular samples $f_i = f(x_i)$
 - Using the Dirac-delta function, we can rewrite f(x) as a convolution

$$f(\mathbf{x}) = \int_{\mathbf{x}'} f(\mathbf{x}') \delta(\|\mathbf{x} - \mathbf{x}'\|) \, dV$$

Replace delta function with a kernel function w_h

$$\tilde{f}(\mathbf{x}) = \int_{\mathbf{x}'} f(\mathbf{x}') \omega_h(\|\mathbf{x} - \mathbf{x}'\|) dV$$
 $\int \omega_h = 1$

Function approximation with SPH

 Discretize the integral into a sum over all sample points to obtain the SPH approximation

$$\tilde{f}(\mathbf{x}) = \int_{\mathbf{x}'} f(\mathbf{x}') \omega_h(\|\mathbf{x} - \mathbf{x}'\|) \, dV \quad \Longrightarrow \quad \langle f \rangle (\mathbf{x}) = \sum_i f_i \omega_h(\|\mathbf{x}_i - \mathbf{x}\|) V_i$$

- How to compute volume V_i for each sample?
 - Associate with mass m_i

$$V_i = \frac{m_i}{\rho_i}$$

- Function approximation with SPH
 - How to compute density estimation?

$$\rho_{i} = \langle \rho \rangle (\mathbf{x}_{i}) = \sum_{j} \omega_{h}(\|\mathbf{x}_{i} - \mathbf{x}_{j}\|) \rho_{j} V_{j} + V_{i} = \frac{m_{i}}{\rho_{i}}$$

$$\rho_{i} = \langle \rho \rangle (\mathbf{x}_{i}) = \sum_{j} \omega_{h}(\|\mathbf{x}_{i} - \mathbf{x}_{j}\|) \rho_{j} V_{j}$$

$$= \sum_{j} \omega_{h}(\|\mathbf{x}_{i} - \mathbf{x}_{j}\|) \rho_{j} \frac{m_{j}}{\rho_{j}}$$

$$= \sum_{j} \omega_{h}(\|\mathbf{x}_{i} - \mathbf{x}_{j}\|) m_{j}$$

Kernel functions

Admissible kernel functions: they must be normalized

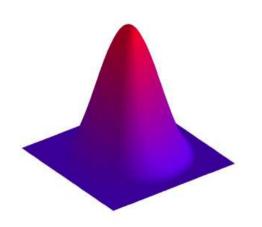
$$\int_{\mathbf{X}} \mathbf{\omega}_h(\|\mathbf{x}\|) \mathrm{d}V = 1$$

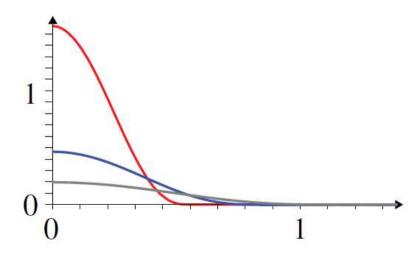
- Smoothing parameter h
 - Allowing control over how far the influence of each sample point reaches (local support)
 - Too large values of h produce unnecessarily smooth reconstructions
 - Kernel function converges to a Dirac-delta function as h goes to zero

Kernel functions

A good polynomial kernel function

$$\omega_h(d) = \begin{cases} \frac{315}{64\pi h^3} \left(1 - \frac{d^2}{h^2}\right)^3 & d < h, \\ 0 & \text{otherwise} \end{cases}$$





Approximation of differential operators

- Apply SPH approximations to the solution of partial differential equations
 - Not only a reconstruction of the continuous function f, but also the derivatives of the function
- Sample values f_i are constants, we can write approximation of gradient as

$$\langle \nabla f \rangle (\mathbf{x}) = \sum_{i} f_{i} \nabla \omega_{h}(\|\mathbf{x} - \mathbf{x}_{i}\|) V_{i}$$

$$\nabla \omega_h(\|\mathbf{x} - \mathbf{x}_i\|) = \frac{\mathbf{x} - \mathbf{x}_i}{\|\mathbf{x} - \mathbf{x}_i\|} \, \omega_h'(\|\mathbf{x} - \mathbf{x}_i\|)$$

- Approximation of differential operators
 - Other linear operators can be treated similarly

$$\langle \Delta f \rangle (\mathbf{x}) = \sum_{i} f_{i} \Delta \omega_{h}(||\mathbf{x} - \mathbf{x}_{i}||) V_{i}$$

$$\langle \nabla \cdot \mathbf{f} \rangle (\mathbf{x}) = \sum_{i} \mathbf{f}_{i} \cdot \nabla \omega_{h}(\|\mathbf{x} - \mathbf{x}_{i}\|) V_{i}$$

- Accuracy of the approximations of derivative
 - Strongly depends on the distribution of sample points within the support region
 - For highly irregular sample distributions, the differential properties can be very noisy

Approximation of differential operators

- Problem with previous estimation
 - Gradient approximation can yield non-zero values even if the function is constant
- How to rectify?
 - Enforce a zero gradient for constant functions by subtracting the constant f_i

$$\nabla f(\mathbf{x}_i) \approx \langle \nabla [f - f_i] \rangle (\mathbf{x}_i)$$

$$= \sum_{j} (f_j - f_i) \nabla \omega_h(||\mathbf{x}_i - \mathbf{x}_j||) V_j$$

- Approximation of differential operators
 - Same reasoning applied to the divergence and Laplace operators

$$\langle \nabla \cdot \mathbf{f} \rangle (\mathbf{x}_i) = \sum_{j} (\mathbf{f}_j - \mathbf{f}_i) \cdot \nabla \omega_h(||\mathbf{x}_i - \mathbf{x}_j||) V_j$$
$$\langle \Delta f \rangle (\mathbf{x}_i) = \sum_{j} (f_j - f_i) \Delta \omega_h(||\mathbf{x}_i - \mathbf{x}_j||) V_j$$

Approximation of differential operators

Another variation of the gradient approximation important in particular for fluid simulation

$$\nabla \left[\frac{f}{\rho} \right] = \frac{\rho \nabla f - f \nabla \rho}{\rho^{2}}$$

$$\Rightarrow \nabla f = \rho \left(\nabla \left[\frac{f}{\rho} \right] + \frac{f \nabla \rho}{\rho^{2}} \right)$$

$$\nabla f(\mathbf{x}_{i}) \approx \rho_{i} \left(\left\langle \nabla \left[\frac{f}{\rho} \right] \right\rangle (\mathbf{x}_{i}) + \frac{f_{i} \left\langle \nabla \rho \right\rangle (\mathbf{x}_{i})}{\rho_{i}^{2}} \right)$$

$$= \rho_{i} \sum_{j} m_{j} \left(\frac{f_{j}}{\rho_{j}^{2}} + \frac{f_{i}}{\rho_{i}^{2}} \right) \nabla \omega_{h} (\|x_{i} - x_{j}\|)$$

For compressible fluids

 The momentum equation can be written as a combination of pressure, viscosity, and external forces

$$\frac{D\mathbf{v}}{Dt} = \frac{1}{\rho} \left(\mathbf{f}_{p} + \mathbf{f}_{v} + \mathbf{f}_{e} \right)$$

- $-\mathbf{f}_{e}$ are external forces acting on the fluid
- The pressure force \mathbf{f}_p is a function of the pressure field p
- Viscous forces smooth the velocity field

$$\mathbf{f}_{p} = -\nabla p$$
$$\mathbf{f}_{v} = \mu \nabla \cdot \nabla \mathbf{v} = \mu \Delta \mathbf{v}$$

- For compressible fluids
 - How to compute pressure
 - Equation of state
 - Common choice: Tait equation

$$p = K\left(\left(\frac{\rho}{\rho_0}\right)^{\gamma} - 1\right)$$

- The most straightforward spatial discretization
 - Consider Lagrangian coordinates

$$\frac{\partial \mathbf{v}_{i}}{\partial t} = -\frac{\langle \nabla p \rangle (\mathbf{x}_{i})}{\rho_{i}} + \mu \langle \Delta \mathbf{v} \rangle (\mathbf{x}_{i}) + \frac{\mathbf{f}_{e}(\mathbf{x}_{i})}{\rho_{i}}$$

Pressure forces

Particle accelerations due to pressure forces

$$\frac{\mathbf{f}_{p}(\mathbf{x}_{i})}{\mathbf{\rho}(\mathbf{x}_{i})} = -\frac{\langle \nabla p \rangle (\mathbf{x}_{i})}{\mathbf{\rho}_{i}}$$

- Implementation

$$\frac{\mathbf{f}_p(\mathbf{x}_i)}{\rho(\mathbf{x}_i)} = -\sum_j \nabla \omega_h^{ij} p_j \frac{m_j}{\rho_j \rho_i} = -\sum_j \mathbf{a}_p^{ji}$$

Pressure forces

- Problem
 - Observation

$$\mathbf{a}_p^{ji} + \mathbf{a}_p^{ij} \neq 0$$

- Linear and angular momentum may not be conserved
- Low resolution problem is more significant (visual artifacts)

Symmetrize

$$\nabla f(\mathbf{x}_{i}) \approx \rho_{i} \left(\left\langle \nabla \left[\frac{f}{\rho} \right] \right\rangle (\mathbf{x}_{i}) + \frac{f_{i} \left\langle \nabla \rho \right\rangle (\mathbf{x}_{i})}{\rho_{i}^{2}} \right)$$

$$= \rho_{i} \sum_{j} m_{j} \left(\frac{f_{j}}{\rho_{j}^{2}} + \frac{f_{i}}{\rho_{i}^{2}} \right) \nabla \omega_{h} (\|x_{i} - x_{j}\|) \qquad \longrightarrow \qquad \frac{\mathbf{f}_{p}(\mathbf{x}_{i})}{\rho(\mathbf{x}_{i})} = -\sum_{j} \nabla \omega_{h}^{ij} (\frac{p_{j}}{\rho_{j}^{2}} + \frac{p_{i}}{\rho_{i}^{2}}) m_{j}$$

Viscosity forces

 Discretizing the acceleration on particles due to viscosity leads to the expression

$$\frac{\mathbf{f}_{v}(\mathbf{x}_{i})}{\rho_{i}} = \mu \sum_{j} \Delta \omega_{h}^{ij} (\mathbf{v}_{j} - \mathbf{v}_{i}) \frac{m_{j}}{\rho_{i} \rho_{j}} = \mu \sum_{j} \mathbf{a}_{v}^{ji}$$

– The force is symmetric:

$$\mathbf{a}_{v}^{ji} + \mathbf{a}_{v}^{ij} = 0$$

Viscosity forces

- Problem
 - Higher-order derivatives depends strongly on the distribution of sample points
 - Quite noisy especially for small support radius of the kernel function
- Smoothing operator
 - Create artificial viscosity

$$\tilde{\mathbf{v}}_{i} = \xi \langle \mathbf{v} \rangle (\mathbf{x}_{i}) + (1 - \xi) \mathbf{v}_{i}
= \xi \left[\sum_{j} \omega_{h}^{ij} \mathbf{v}_{j} \frac{m_{j}}{\rho_{j}} \right] + (1 - \xi) \mathbf{v}_{i}$$

$$0 \le \xi \le 1$$

- Time discretization and simulation loop
 - Using a simple explicit integration scheme

$$\mathbf{v}_{i}(t + \Delta t) = \mathbf{v}_{i}(t) + \Delta t \frac{D\mathbf{v}_{i}}{Dt}$$

$$\mathbf{x}_{i}(t + \Delta t) = \mathbf{x}_{i}(t) + \Delta t \mathbf{v}_{i}(t + \Delta t)$$

Higher order time-integrator can be applied

- Prediction-correction scheme
 - PCISPH, Solenthaler et al. SIGGRAPH 2009
 - The density at a point in time t + 1

$$\rho_{i}(t+1) = m \sum_{j} W(\mathbf{x}_{i}(t+1) - \mathbf{x}_{j}(t+1))$$

$$= m \sum_{j} W(\mathbf{x}_{i}(t) + \Delta \mathbf{x}_{i}(t) - \mathbf{x}_{j}(t) - \Delta \mathbf{x}_{j}(t))$$

$$= m \sum_{j} W(\mathbf{d}_{ij}(t) + \Delta \mathbf{d}_{ij}(t))$$
where $\mathbf{d}_{ij}(t) = \mathbf{x}_{i}(t) - \mathbf{x}_{j}(t)$, and $\Delta \mathbf{d}_{ij}(t) = \Delta \mathbf{x}_{i}(t) - \Delta \mathbf{x}_{j}(t)$

Prediction-correction scheme

– Assuming that Δd_{ij} is relatively small, the first order Taylor approximation can be applied

$$\rho_{i}(t+1) = m \sum_{j} W(\mathbf{d}_{ij}(t)) + \nabla W(\mathbf{d}_{ij}(t)) \cdot \Delta \mathbf{d}_{ij}(t)$$

$$= m \sum_{j} W(\mathbf{x}_{i}(t) - \mathbf{x}_{j}(t)) +$$

$$m \sum_{j} \nabla W(\mathbf{x}_{i}(t) - \mathbf{x}_{j}(t)) \cdot (\Delta \mathbf{x}_{i}(t) - \Delta \mathbf{x}_{j}(t))$$

$$= \rho_{i}(t) + \Delta \rho_{i}(t).$$

• The term $\Delta \rho_i(t)$ is unknown, a function of p which we are looking for

- Prediction-correction scheme
 - Density error predictor

$$\rho_{err_i}^* = \rho_i^* - \rho_0$$

Pressure corrector

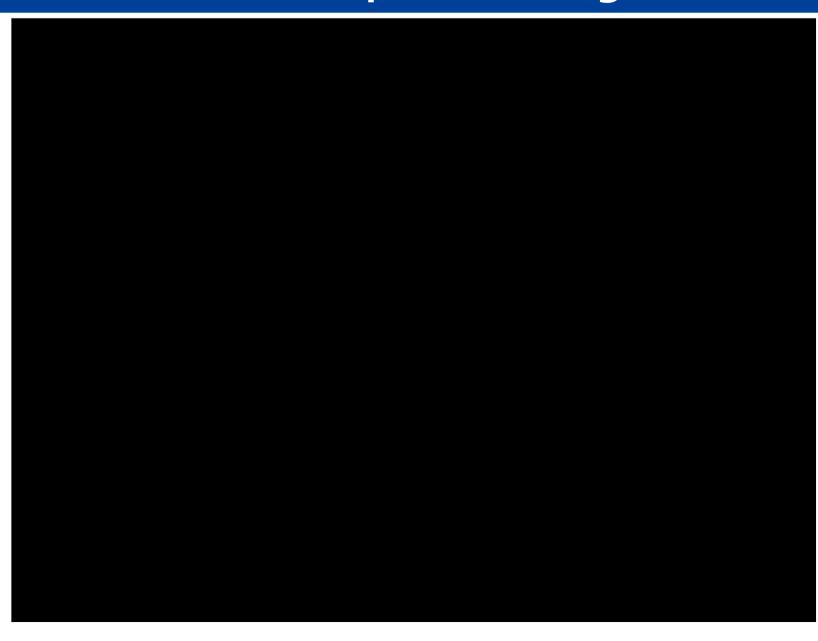
$$\delta = \frac{-1}{\beta(-\sum_{j} \nabla W_{ij} \cdot \sum_{j} \nabla W_{ij} - \sum_{j} (\nabla W_{ij} \cdot \nabla W_{ij}))}$$

$$\tilde{p}_i = \delta \rho_{err_i}^*$$

$$p_i += \tilde{p}_i$$



Particle solver coupled with rigid bodies



End of the lecture