

Pole-Placement Design – A Polynomial Approach

Overview

- A Simple Design Problem
- The Diophantine Equation
- More Realistic Assumptions

A Simple Design Problem

- System:

$$A(q)y[k] = B(q)u[k] \quad (1)$$

- Assumptions:

- $A = q^{n_a} + a_1 q^{n_a-1} + \dots + a_{n_a-1} q + a_{n_a}$ is monic
 - $\deg A > \deg B$.
 - Disturbances are widely spread impulses
- Specifications are given by the closed-loop characteristic polynomial, and the controller may have certain properties, for example, integral action
- General linear controller (u_c : command signal, y : measured output, u : control signal):

$$R(q)u[k] = T(q)u_c[k] - S(q)y[k] \quad (2)$$

- $R(q)$ is chosen to be monic.
- Feedforward part:

$$H_{ff}(z) = \frac{T(z)}{R(z)} \quad (3)$$

- Feedback part:

$$H_{fb}(z) = \frac{S(z)}{R(z)} \quad (4)$$

- Goal: Causal controller with no time-delay \Rightarrow

$$\deg R = \deg S \quad (5)$$

Solving the design process

- Eliminating $u[k]$ between the process model (1) and the controller (2) gives

$$(A(q)R(q) + B(q)S(q)) y[k] = B(q)T(q)u_c[k] \quad (6)$$

- Closed-loop characteristic polynomial:

$$A_{cl}(z) = A(z)R(z) + B(z)S(z) \quad (7)$$

- The pole-placement design is to find polynomials S and R that satisfy Eq. (7) for given A , B , and A_{cl} .
- Eq. (7) is called *Diophantine Equation*.
- Factorising the A_{cl} polynomial:

$$A_{cl}(z) = A_c(z)A_o(z) \quad (8)$$

- We call $A_c(z)$ the *controller polynomial* and $A_o(z)$ the *observer polynomial*.
- In order to determine the polynomial T , we calculate the pulse-transfer function from the command signal to the output:

$$Y(z) = \frac{B(z)T(z)}{A_{cl}(z)}U_c(z) = \frac{B(z)T(z)}{A_c(z)A_o(z)}U_c(z) \quad (9)$$

- Zeros of the open-loop system are also zeros of the closed-loop system (unless $B(z)$ and $A_{cl}(z)$ have common factors).
- Let's choose the polynomial T so that it cancels the observer polynomial A_o :

$$T(z) = t_0 A_o(z) \quad (10)$$

- The response to command signals is then given by

$$Y(z) = \frac{t_0 B(z)}{A_c(z)}U_c(z) \quad (11)$$

where t_0 is chosen to obtain a desired static gain for the system (e.g., for unit gain:

$$t_0 = A_c(1)/B(1)).$$

The Diophantine Equation - Minimal-Degree Solution:

- The equation

$$A_{cl}(z) = A(z)R(z) + B(z)S(z) \quad (12)$$

has a solution only if the greatest common divisor of A and B divides A_{cl} .

- Number of controller parameters when $\deg R = \deg S$:

$$n_p = 2(\deg R + 1) = 2 \deg R + 2 \quad (13)$$

- Degree of A_{cl} (remind that $\deg(AR) > \deg(BS)$):

$$\deg(A_{cl}) = \max(\deg(AR), \deg(BS)) = \deg(AR) = \deg(A) + \deg(R) \quad (14)$$

- Number of equations (coefficients of A_{cl}):

$$n_e = \deg(A_{cl}) + 1 = \deg(A) + \deg(R) + 1 \quad (15)$$

- Unique minimal solution:

$$n_e = n_p \quad (16)$$

$$\deg A + \deg R + 1 = 2 \deg R + 2 \quad (17)$$

\Rightarrow

$$\deg R = \deg A - 1 \quad (18)$$

- Degree of the closed-loop polynomial for the minimum-degree solution:

$$\deg A_{cl} = 2 \deg(A) - 1 \quad (19)$$

- The Diophantine equation can be solved using matrix calculations ($n = n_a > n_b$):

$$\begin{pmatrix}
 \bar{a}_0 & 0 & 0 & \cdots & 0 & \bar{b}_0 & 0 & 0 & \cdots & 0 \\
 \bar{a}_1 & \bar{a}_0 & 0 & \cdots & 0 & \bar{b}_1 & \bar{b}_0 & 0 & \cdots & 0 \\
 \bar{a}_2 & \bar{a}_1 & \bar{a}_0 & \cdots & 0 & \bar{b}_2 & \bar{b}_1 & \bar{b}_0 & \cdots & 0 \\
 \vdots & \vdots & \vdots & \ddots & \vdots & \vdots & \vdots & \vdots & \ddots & \vdots \\
 \bar{a}_n & \bar{a}_{n-1} & \bar{a}_{n-2} & \cdots & \bar{a}_0 & \bar{b}_n & \bar{b}_{n-1} & \bar{b}_{n-2} & \cdots & \bar{b}_0 \\
 0 & \bar{a}_n & \bar{a}_{n-1} & \cdots & \bar{a}_1 & 0 & \bar{b}_n & \bar{b}_{n-1} & \cdots & \bar{b}_1 \\
 0 & 0 & \bar{a}_n & \cdots & \bar{a}_2 & 0 & 0 & \bar{b}_n & \cdots & \bar{b}_2 \\
 \vdots & \vdots & \vdots & \ddots & \vdots & \vdots & \vdots & \vdots & \ddots & \vdots \\
 0 & 0 & 0 & \cdots & \bar{a}_n & 0 & 0 & 0 & \cdots & \bar{b}_n
 \end{pmatrix}
 \begin{pmatrix}
 r_0 \\
 r_1 \\
 \vdots \\
 r_{n_a-1} \\
 s_0 \\
 \vdots \\
 s_{n_a-1}
 \end{pmatrix}
 =
 \begin{pmatrix}
 a_{cl,0} \\
 a_{cl,1} \\
 a_{cl,2} \\
 \vdots \\
 a_{cl,n} \\
 a_{cl,n+1} \\
 a_{cl,n+2} \\
 \vdots \\
 a_{cl,2n-1}
 \end{pmatrix}$$

$$\bar{B} = \bar{b}_0 q^n + \bar{b}_1 q^{n-1} + \cdots + \bar{b}_{n-1} q + \bar{b}_n = b_0 q^{n_b} + b_1 q^{n_b-1} + \cdots + b_{n_b-1} q + b_{n_b} \quad (21)$$

$$\bar{A} = \bar{a}_0 q^n + \bar{a}_1 q^{n-1} + \cdots + \bar{a}_{n-1} q + \bar{a}_n = 1 q^{n_a} + a_1 q^{n_a-1} + \cdots + a_{n_a-1} q + a_{n_a} \quad (22)$$

More Realistic Assumptions*Cancellations of Poles and Zeros*

- Factorisation of A and B:

$$A = A^+ A^- \quad (23)$$

$$B = B^+ B^- \quad (24)$$

- A^+ and B^+ are stable (well damped) factors that can be cancelled (both should be chosen monic).
- Poles that shall be cancelled must be controller zeros and zeros that must be cancelled must be controller poles:

$$R = B^+ R_d \bar{R} \quad (25)$$

$$S = A^+ S_d \bar{S} \quad (26)$$

where R_d and S_d are fixed pre-determined parts of the controller (see next subsection).

- Closed-loop polynomial:

$$A_{cl} = AR + BS = A^+ B^+ (R_d \bar{R} A^- + S_d \bar{S} B^-) = A^+ B^+ \bar{A}_{cl} \quad (27)$$

- Cancelled zeros and poles are part of the closed-loop polynomial and must therefore be well damped!
- Cancelling the common factors we find that the polynomials \bar{R} and \bar{S} must satisfy:

$$\bar{R} R_d A^- + \bar{S} S_d B^- = \bar{A}_{cl} \quad (28)$$

- Minimal-degree solution:

$$n_e = n_p \quad (29)$$

$$\deg \bar{A}_{cl} + 1 = \deg \bar{R} + \deg \bar{S} + 2 \quad (30)$$

$$\max(\deg \bar{R} R_d A^-, \deg \bar{S} S_d B^-) + 1 = \deg \bar{R} + \deg \bar{S} + 2 \quad (31)$$

- For $\deg \bar{R} R_d A^- > \deg \bar{S} S_d B^-$ we obtain:

$$\deg \bar{S} = \deg A^- + \deg R_d - 1 \quad (32)$$

$$\deg(\bar{R} R_d A^-) = \max(\deg \bar{A}_{cl}, \deg \bar{S} S_d B^-) = \deg \bar{A}_{cl} \quad (33)$$

$$\deg S = \deg R \quad (34)$$

$$\underbrace{\deg A^- + \deg R_d - 1}_{\deg \bar{S}} + \deg S_d + \deg A^+ =$$

$$\underbrace{\deg \bar{A}_{cl} - \deg A^- - \deg R_d}_{\deg \bar{R}} + \deg R_d + \deg B^+ \quad (35)$$

Solving this equation for $\deg \bar{A}_{cl}$ yields:

$$\deg \bar{A}_{cl} = 2 \deg A + \deg R_d + \deg S_d - \deg B^+ - \deg A^+ - 1 \quad (36)$$

$$\deg A_{cl} = 2 \deg A + \deg R_d + \deg S_d - 1 \quad (37)$$

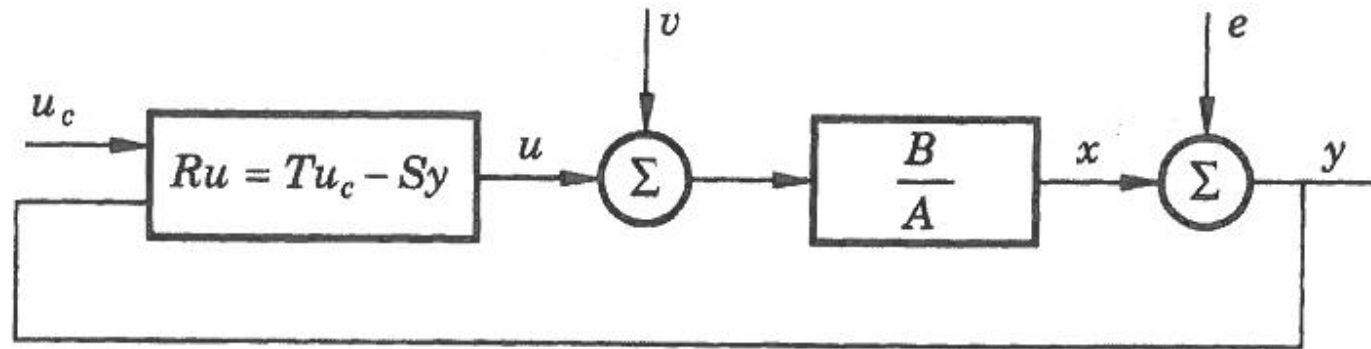
- In order to solve Eq. (28) we create polynomials \bar{A} and \bar{B} with $\deg \bar{A} = \deg \bar{B} = n = \max(\deg(A^- R_d), \deg(B^- S_d))$:

$$\bar{A} = A^- R_d \quad (38)$$

$$\bar{B} = B^- S_d \quad (39)$$

$$\deg \bar{A}_{cl} + 1 \left\{ \underbrace{\begin{pmatrix} \ddots & \vdots & \vdots & \ddots & \vdots & \vdots \\ \cdots & \bar{a}_{\deg \bar{A}} & \bar{a}_{\deg \bar{A}-1} & \cdots & \bar{b}_{\deg \bar{B}} & \bar{a}_{\deg \bar{B}-1} \\ \cdots & 0 & \bar{a}_{\deg \bar{A}} & \cdots & 0 & \bar{b}_{\deg \bar{B}} \end{pmatrix}}_{\deg \bar{R}+1} \underbrace{\begin{pmatrix} \ddots & \vdots & \vdots \\ \cdots & \bar{b}_{\deg \bar{B}} & \bar{a}_{\deg \bar{B}-1} \\ \cdots & 0 & \bar{b}_{\deg \bar{B}} \end{pmatrix}}_{\deg \bar{S}+1} \begin{pmatrix} \bar{r}_0 \\ \vdots \\ \bar{s}_0 \\ \vdots \end{pmatrix} = \begin{pmatrix} \bar{a}_{cl_0} \\ \bar{a}_{cl_1} \\ \vdots \end{pmatrix} \quad (40)$$

Handling disturbances



$$x = \frac{BT}{AR + SB}u_c + \frac{BR}{AR + SB}v - \frac{BS}{AR + SB}e$$

$$y = \frac{BT}{AR + SB}u_c + \frac{BR}{AR + SB}v + \frac{AR}{AR + SB}e$$

$$u = \frac{AT}{AR + SB}u_c - \frac{BS}{AR + SB}v - \frac{AS}{AR + SB}e$$

- To avoid steady-state errors due to constant load disturbances the static gain from the disturbance v to y must be zero:

$$B(1)R(1) = 0$$

If $B(1) \neq 0$ then we must require that $R(1) = 0$. This means that $R_d = z - 1$ is a factor of $R(z)$ or that the controller is required to have integral action.

- Elimination of periodic load disturbances (with period $n \cdot \Delta$) by using $R_d = z^n - 1$:

$$v((k + n)\Delta) - v(k\Delta) = (z^n - 1)v(k\Delta) = 0$$

- Elimination of sinusoidal load disturbances with frequency ω_0 :

$$R_d = z^{-2} + z \cos(\omega_0 \Delta) + 1$$

- Eliminating the effect of measurement noise at Nyquist frequency:

$$S_d = z + 1$$

Pre-filter revisited

- Let us factorise the polynomial $A_{cl} = \underbrace{A^+ \bar{A}_o}_{A_o} \underbrace{B^+ \bar{A}_c}_{A_c}$
- Let's choose the polynomial T so that it cancels the observer polynomial A_o :

$$T(z) = t_0 A_o(z) \quad (41)$$

- The response to command signals is then given by

$$Y(z) = \frac{t_0 B(z)}{A_c(z)} U_c(z) = H_m(z) U_c(z) \quad (42)$$

where t_0 is chosen to obtain a desired static gain for the system (e.g., for unit gain: $t_0 = A_c(1)/B(1)$).