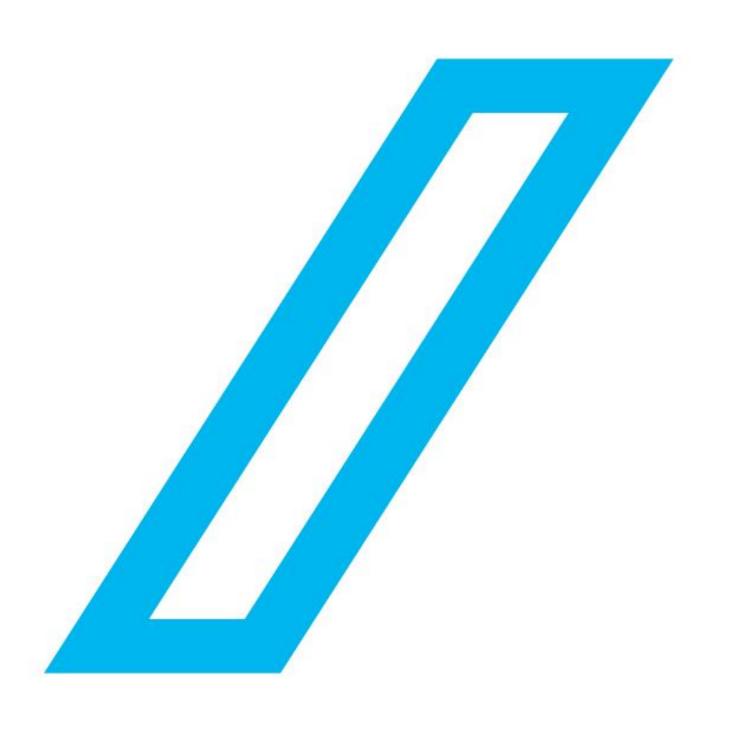


Vehicle model

Lecturer: Seungmok Song



Contents

- 1. Introduction
- 2. Longitudinal vehicle model
- 3. Lateral vehicle model

Introduction

- Vehicle model
 - 자동차를 처음 운전할 때 가장 먼저 해보는 것
 - Steering, brake, throttle을 작동시켰을 때 어떻게 차가 움직이는지 감을 익힌다
 - 머리속에 input 대비 output에 대한 모델을 만드는 것이 목적





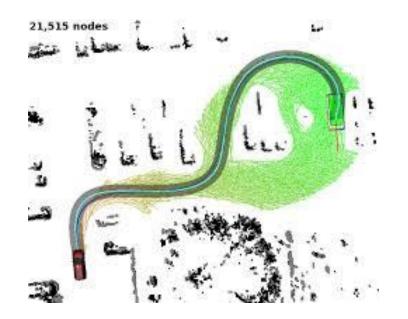
Introduction

- Motion planning and control for autonomous vehicle
 - 차량의 steering, brake, throttle 대비 차량이 움직이는 수학적 모델이 기초
 - From kinematic model to dynamic model
 - Kinematic model: force, torque의 영향을 무시하고 기구학적인 요소만 고려
 - Dynamic model: Kinematic model보다 높은 정확도 제공, 공기저항이나 road friction, slip 등이 더 고려됨

https://youtu.be/WNIDcT0Zdj4?si=0stZ GJ-VyQBzrS6

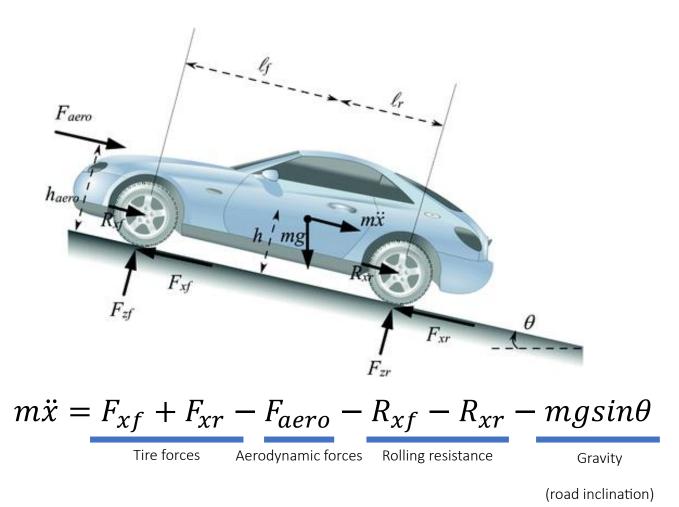
https://youtu.be/3x3SqeSdrAE?si=2yyvn64GrfOXhBj0

Paper: https://ddl.stanford.edu/sites/g/files/sbiybj9456/f/marty_avec2018_fullpaper.pdf





- Dynamic model
 - force balance on a vehicle



HL Klemove

- Aerodynamics
 - Exact model of air flow is too complicated
 - Semi-empirical drag force model is used

$$m\ddot{x} = F_{xf} + F_{xr} - F_{aero} - R_{xf} - R_{xr} - mgsin\theta$$

$$F_{aero} = \frac{1}{2} \rho \underline{V^2} C_D A$$
Dominant in high speed!

- ρ : air density
- C_D : drag coefficient
- *V*: relative velocity of the vehicle (with respect to the wind)
- A: Frontal aera of the vehicle



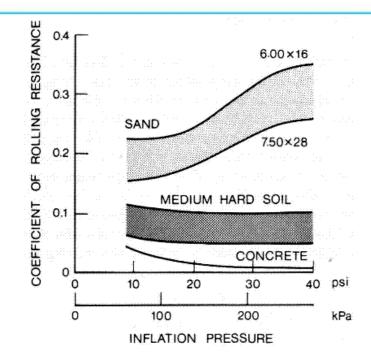
			CL	C_D
1	Low drag body of revolution		0	0.04
2	Low drag vehicle near the ground	ramanna.	0.18	0.15
3	Generic automobile		0.28	0.35
4	Prototype race car		-3.00	0.75

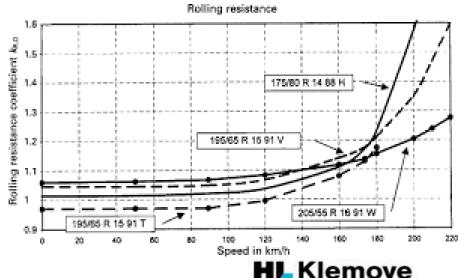
- Rolling resistance
 - The total rolling resistance is the sum of the resistances from all the wheels
 - Tire inflation pressure: depends on surface
 - Speed: the effect is small at low speed → assumed to be constant

$$m\ddot{x} = F_{xf} + F_{xr} - F_{aero} - R_{xf} - R_{xr} - mgsin\theta$$

$$R_{x} = R_{xf} + R_{xr} = f_{r}W$$

- f_r : rolling resistance coefficient
- W: wight of the vehicle

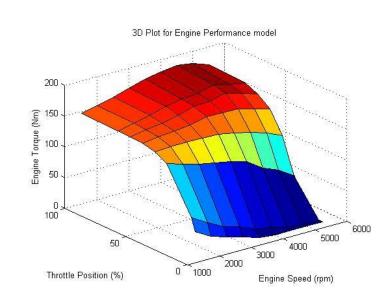


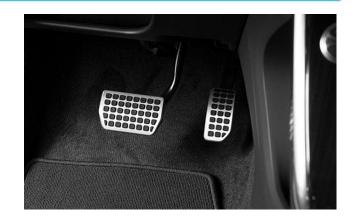


- Tire forces: traction & brake forces
 - Powertrain: throttle and brake system
 - https://youtu.be/aS615xkzmfs (참고)

$$m\ddot{x} = F_{xf} + F_{xr} - F_{aero} - R_{xf} - R_{xr} - mgsin\theta$$

 $\tau = f(throttle, brake, rpm, \cdots)$

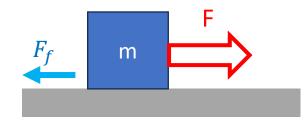






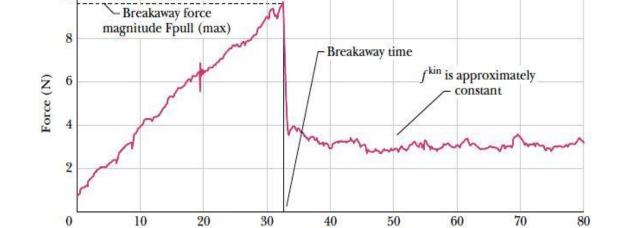


- Tire forces: traction & brake forces
 - Friction



$$F_{f,S} = \mu_S N$$

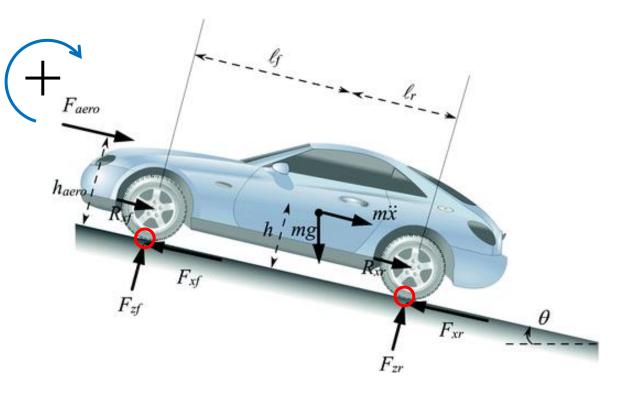
$$F_{f,m} = \mu_m N$$



Time (s)
Source: Cummings, Laws, Redish, & Cooney, Understanding Physics (Wiley, 2003)

$$F_{f,S} = \mu_S N = \mu_S F_Z$$

- Tire forces : traction & brake forces
 - Dynamic axle loads on a vehicle
 - Rotational system on each tire



Front wheel

$$h_{aero}F_{aero} + mgl_f\cos(\theta) + mgh\sin(\theta) + mh\ddot{x} - F_{zr}L = 0$$

Rear wheel

$$h_{aero}F_{aero} - mgl_r\cos(\theta) + mgh\sin(\theta) + mh\ddot{x} + F_{zf}L = 0$$

$$F_{zf} = \frac{mgl_r \cos(\theta) - mgh \sin(\theta) - h_{aero} F_{aero} - mh\ddot{x}}{L}$$

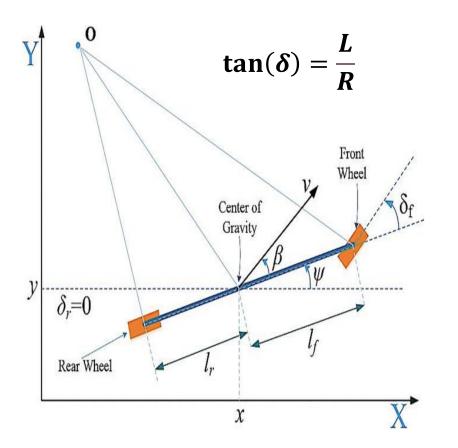
$$F_{zr} = \frac{mgl_f \cos(\theta) + mgh \sin(\theta) + h_{aero} F_{aero} + mh\ddot{x}}{L}$$

$$F_{zf} = \frac{mgl_r - h_{aero}F_{aero} - mh\ddot{x}}{L}$$

$$F_{zr} = \frac{mgl_f + h_{aero}F_{aero} + mh\ddot{x}}{L}$$



- Kinematic bicycle model
 - 구조적인 특성만을 이용한 모델
 - Force 를 사용하지 않음
 - 각 전륜 / 후륜을 하나의 바퀴로 근사(자전거)
 - No tire slip



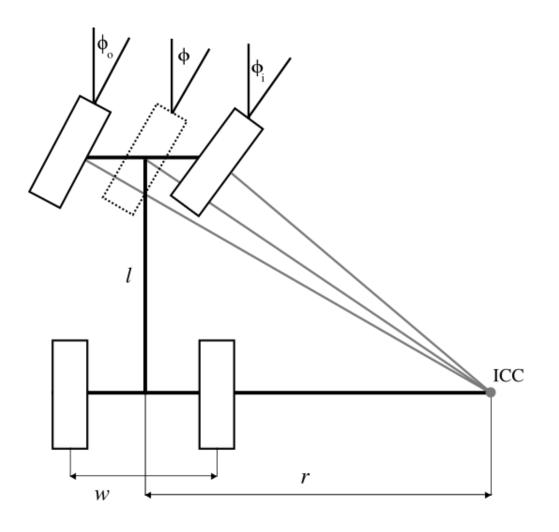
$$\dot{\psi} = \frac{V_x}{R} = V_x \left(\frac{\tan(\delta)}{L}\right) = \frac{V_x \tan(\delta)}{L}$$
 Rear wheel
$$\dot{X} = V_x \cos(\psi)$$

$$\dot{Y} = V_x \sin(\psi)$$

$$\dot{\psi} = \frac{V_{x}\cos(\beta)}{R_{c} \approx R} = V_{x}\cos(\beta) \left(\frac{\tan(\delta)}{L}\right) = \frac{V_{x}\cos(\beta)\tan(\delta)}{L}$$
Center of mass
$$\dot{X} = V_{x}\cos(\psi + \beta)$$

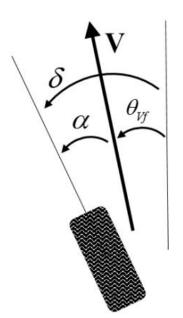
$$\dot{Y} = V_{x}\sin(\psi + \beta)$$

Ackermann steering geometry



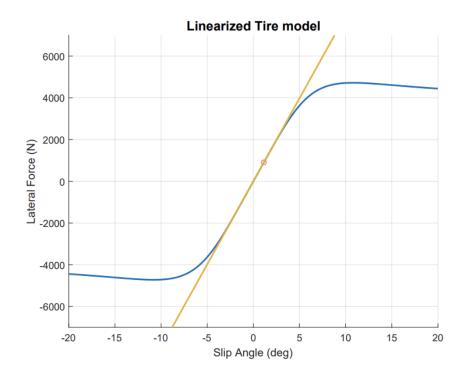
$$\tan(\delta_l) = \frac{l}{R + \frac{w}{2}}$$
$$\tan(\delta_r) = \frac{l}{R + \frac{w}{2}}$$

- Dynamic bicycle model
 - Tire forces : 횡방향(y) 힘의 근원지!
 - Tire slip : 타이어 방향과 실제 움직이는 방향(Heading angle)과의 차이 (Heading angle → Tire direction)



$$\alpha = \delta - \theta_v$$

$$F_y = C_\alpha \alpha = C_\alpha (\delta - \theta_v)$$
 C_α : Cornering stiffness



- Dynamic bicycle model
 - Bicycle model formulation
 - With small steering angle(δ)

$$ma_{y} = F_{yf} + F_{yr}$$
$$I\ddot{\psi} = l_{f}F_{yf} - l_{r}F_{yr}$$

$$F_{yf} = 2C_{\alpha f}\alpha_f \cos(\delta)$$

$$F_{yr} = 2C_{\alpha r}\alpha_r$$

$$\delta - \theta_f = \alpha_f$$

$$-\theta_r = \alpha_r$$

$$F_{yr}$$

$$\delta = \frac{1}{l_r}$$

$$\tan(\theta_f) = \frac{v_y + \dot{\psi}l_f}{v_x} \approx \theta_f$$

$$\tan(\theta_r) = \frac{v_y - \dot{\psi}l_r}{v_x} \approx \theta_r$$

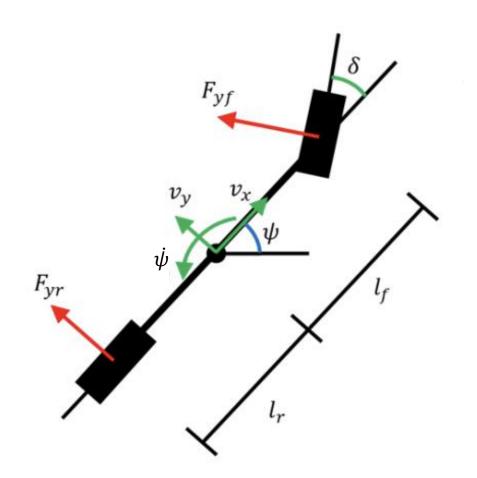


- Dynamic bicycle model
 - Acceleration of moving vehicle

$$a_y = \ddot{y} + \dot{\psi}^2 R$$

• Dynamic equations

$$m(\ddot{y} + \dot{\psi}^2 R) = 2C_{\alpha f}\alpha_f + 2C_{\alpha r}\alpha_r$$
$$I\ddot{\psi} = l_f F_{yf} - l_r F_{yr}$$

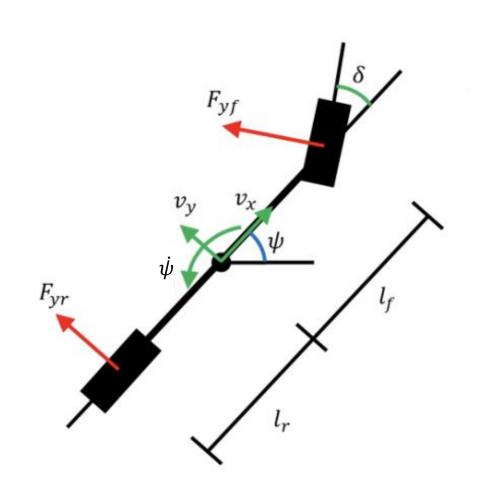


Dynamic bicycle model

$$\ddot{y} = \left(-\frac{2C_{\alpha f} + 2C_{\alpha r}}{mv_{x}}\right)\dot{y} + \left(-v_{x} - \frac{2C_{\alpha f}l_{f} - 2C_{\alpha r}l_{r}}{mv_{x}}\right)\dot{\psi} + \frac{2C_{\alpha f}}{m}\delta$$

$$\ddot{\psi} = \left(-\frac{2C_{\alpha f}l_f - 2C_{\alpha r}l_r}{Iv_{\chi}}\right)\dot{y} - \left(\frac{2C_{\alpha f}l_f^2 + 2C_{\alpha r}l_r^2}{Iv_{\chi}}\right)\dot{\psi} + \frac{2C_{\alpha f}l_f}{I}$$

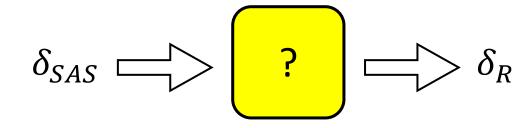
$$\frac{d}{dt} \begin{pmatrix} \dot{y} \\ \dot{\psi} \\ \dot{\psi} \end{pmatrix} = \begin{pmatrix} 0 & 1 & 0 & 0 \\ 0 & -\frac{2C_{\alpha f} + 2C_{\alpha r}}{mv_{x}} & 0 & -v_{x} - \frac{2C_{\alpha f}l_{f} - 2C_{\alpha r}l_{r}}{mv_{x}} \\ 0 & 0 & 0 & 1 \\ 0 & -\frac{2C_{\alpha f}l_{f} - 2C_{\alpha r}l_{r}}{Iv_{x}} & 0 & -\frac{2C_{\alpha f}l_{f}^{2} + 2C_{\alpha r}l_{r}^{2}}{Iv_{x}} \end{pmatrix} \begin{pmatrix} \dot{y} \\ \dot{\psi} \\ \dot{\psi} \end{pmatrix} + \begin{pmatrix} \frac{2C_{\alpha f}}{m} \\ 0 \\ \frac{2C_{\alpha f}l_{f}}{I} \end{pmatrix} \delta$$

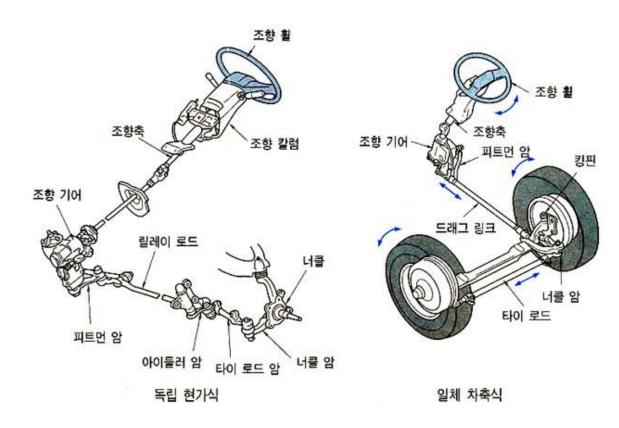


• Steering wheel angle, road wheel angle

 δ_{SAS} : Steering wheel angle(from steering wheel sensor)

 δ_R : Road wheel angle





http://jwkang7.wo.to/pds03/317.htm



Thank You

