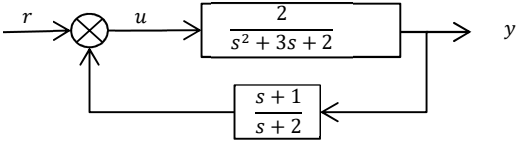
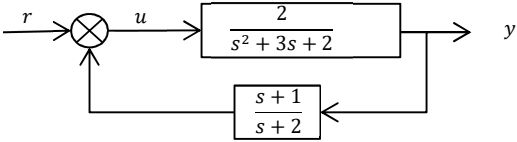


Representation and Analysis of Dynamical Systems

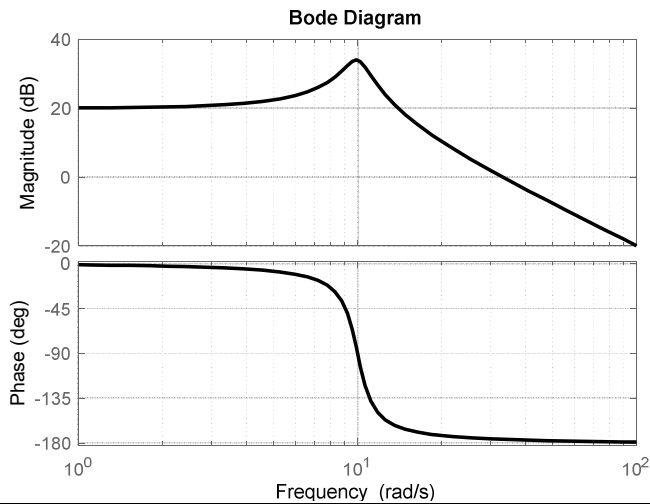
Test – 40min – without documentation

$u(t)$ is unitary step signal	1. The Laplace transform $U(s)$ of $u(t)$ is: a : $U(s) = 1$ b : $U(s) = 1/s$	
The Laplace transform of a signal $u(t)$ is: $U(s) = \frac{10}{10 + s}$	2. The final value (at $t = \infty$) of $u(t)$ is: a : 10 b : 1 c : 0	
The transfer function of a system is: $U(s) = \frac{10}{10 + s}$	3. The static gain is: a : 10 b : 1 c : 0	
A linear system (input $u(t)$, output $y(t)$) is driven by the differential equation: $y''(t) - 3y'(t) + 2y(t) = u(t)$	4. The transfer function is: a : $F(s) = \frac{1}{s^2 - 3s + 2}$ b : $F(s) = \frac{1}{2s^2 - 3s + 1}$	
The system (input $u(t)$, output $y(t)$) driven by the following equation $y''(t) + 3y'(t) - 2y(t) = u(t)$ Is stable	5. The system is stable: a : True b : False	
A linear system (input $u(t)$, output $y(t)$) is driven by the differential equation: $y''(t) + 3y'(t) + 2y(t) = u(t) + u'(t)$	6. The transfer function is: $F(s) = \frac{1}{s + 2}$ a : True b : False	
Consider the system: 	7. The transfer function between r and y is: $\frac{y}{r} = \frac{2s + 2}{s^3 + 5s^2 + 10s + 6}$ a : True b : False	
Consider the system: 	8. The transfer function between r and u is: $\frac{u}{r} = \frac{s^2 + 4s + 4}{s^2 + 4s + 6}$ a : True b : False	
Consider the transfer function of a system: $F(s) = \frac{1}{s}$	9. The system can be stabilized with a pure proportional controller: a : Yes b : No	

The transfer function of a system is:

$$F(s) = \frac{A}{\omega_0^2 + 2\sigma\omega_0 s + s^2}$$

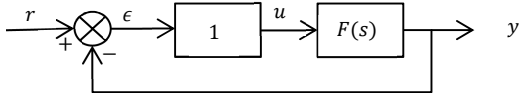
The Bode plot is given below:



10. The correct set of coefficients is:

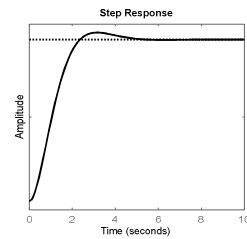
- a: $A = 1 ; \sigma = 0.1 ; \omega_0 = 10$
- b: $A = 1000 ; \sigma = 0.1 ; \omega_0 = 1$
- c: $A = 10 ; \sigma = 10 ; \omega_0 = 1$
- d: $A = 1000 ; \sigma = 0.1 ; \omega_0 = 10$
- e: none

The system given by the last question is included in a closed loop such as:

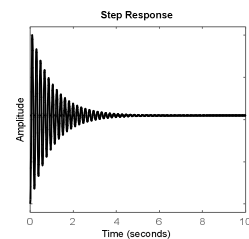


11. The correct step response (input $r(t)$ is a step) is:

a:



b:



Consider the transfer function of a system:

$$F(s) = \frac{1}{1-s}$$

12. The system is stable

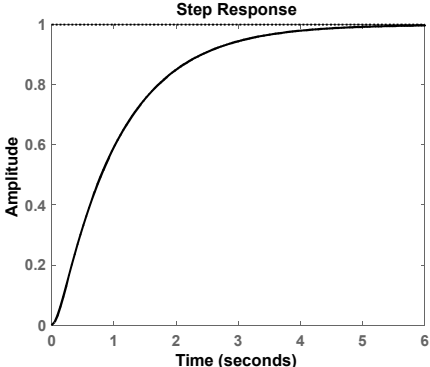
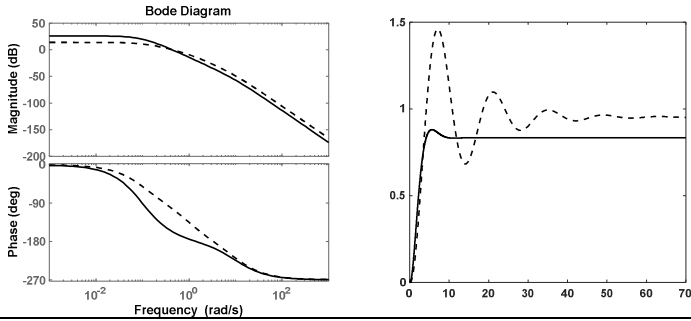
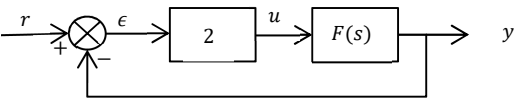
- a: Yes
- b: No

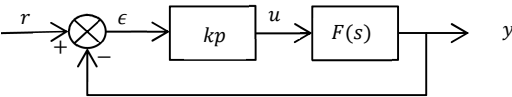
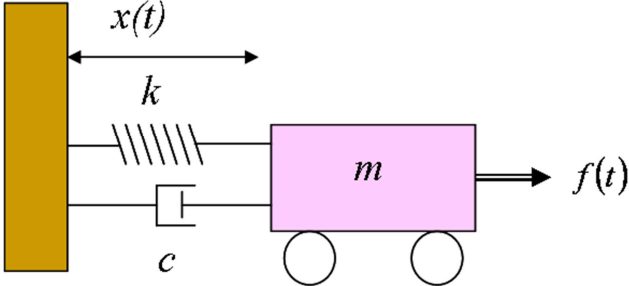
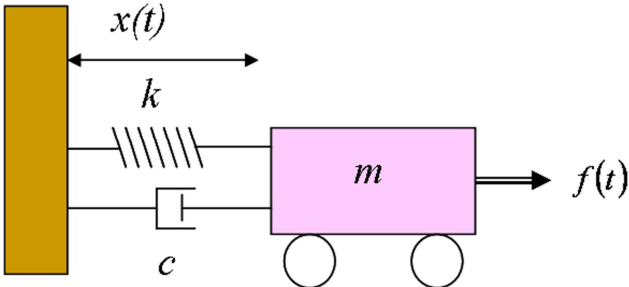
Consider the transfer function of a system:

$$F(s) = \frac{1}{1-s}$$

13. The system can be stabilized with a pure proportional controller:

- a: Yes
- b: No

<p>The step response of a system is given below:</p> 	<p>14. This step response correspond to the transfer function:</p> <p>a: $F_1 = \frac{1}{1+0.1s}$</p> <p>b: $F_1 = \frac{10}{1+s}$</p> <p>c: $F_1 = \frac{1}{1+1.1s+0.1s^2}$</p> <p>d: $F_1 = \frac{1}{1+0.1s+0.1s^2}$</p>	
<p>We give the Bode diagram (open loop) and step response (in closed loop) of two systems (solid line and dashed line):</p> 	<p>15. The system with dashed line (resp. solid) of the Bode diagram corresponds to the system with dashed line (resp. solid) of the step response:</p> <p>a: Yes</p> <p>b: No</p>	
<p>The open loop transfer function H is:</p> $H(s) = \frac{100}{s^2 + 4s}$	<p>16. This system in closed loop has a zero static error:</p> <p>a: Yes</p> <p>b: No</p>	
<p>A system is given by its transfer function:</p> $F(s) = \frac{1-s}{1+s+s^2}$ 	<p>17. This system is stable in open loop (input $u(t)$ output $y(t)$):</p> <p>a: Yes</p> <p>b: No</p> <p>18. This system is stable in closed loop (input $r(t)$ output $y(t)$):</p> <p>a: Yes</p> <p>b: No</p>	
<p>A system (input $u(t)$ output $y(t)$) is driven by the differential equation:</p> $y''(t) + 2y'(t) + y^2(t) = u'(t) + u(t)$ <p>The Laplace transform of $u(t)$ (resp $y(t)$) is $U(s)$ (resp $Y(s)$)</p>	<p>19. The relationship between $U(s)$ and $Y(s)$ is:</p> $s^2Y(s) + 2sY(s) + Y(s)^2 = sU(s) + U(s)$ <p>a: True</p> <p>b: False</p>	

<p>A system (input $u(t)$ output $y(t)$ internal state $x(t)$) is driven by the state space equation:</p> $\begin{cases} \dot{x}(t) = -x(t)^2 + u(t) \\ y(t) = x(t)^2 \end{cases}$ <p>The state and output variation near the equilibrium point (U_0, X_0, Y_0) are $(\delta u(t), \delta x(t), \delta y(t))$ such as:</p> $\begin{aligned} u(t) &= U_0 + \delta u(t) \\ x(t) &= X_0 + \delta x(t) \\ y(t) &= Y_0 + \delta y(t) \end{aligned}$ <p>The system is trimmed at the equilibrium point corresponding to $u(t) = U_0 = 4$.</p>	<p>20. The linearized state space equation near the equilibrium point is:</p> <p>a: $\begin{cases} \dot{\delta x}(t) = -4\delta x(t) + \delta u(t) \\ \delta y(t) = 4\delta x(t) \end{cases}$</p> <p>b: $\begin{cases} \dot{\delta x}(t) = -2\delta x(t) + \delta u(t) \\ \delta y(t) = 2\delta x(t) \end{cases}$</p> <p>c: $\begin{cases} \dot{\delta x}(t) = 4\delta x(t) + \delta u(t) \\ \delta y(t) = -4\delta x(t) \end{cases}$</p>
<p>A system is given by its transfer function:</p> $F(s) = \frac{2-s}{-1+s+s^2}$ 	<p>21. Which one of the following is true:</p> <p>a: closed loop stable if $kp < 0$</p> <p>b: closed loop stable if $kp > 1$</p> <p>c: closed loop stable if $kp = 0.7$</p> <p>d: unstable for any value of kp</p>
	<p>22. The potential energy is</p> <p>a: $E_p = \frac{1}{2}kx^2$</p> <p>b: $E_p = \frac{1}{2}m\dot{x}^2$</p> <p>c: $E_p = 0$</p>
<p>State Space representation</p> <p>Consider the differential equation that characterizes a mechanical system with a mass M, a damper of constant C and a stiffness K:</p> $m\ddot{x} + c\dot{x} + kx = F,$ <p>F being the force applied to the system.</p> <p>The output of the system is the displacement x.</p> 	<p>23. A possible representation of the mechanical system is:</p> <p>a:</p> $\begin{aligned} X &= [x \quad \dot{x}]^t \\ \dot{X} &= \begin{bmatrix} 0 & 1 \\ \frac{k}{m} & \frac{c}{m} \end{bmatrix} X + \begin{bmatrix} 0 \\ \frac{1}{m} \end{bmatrix} F \\ Y &= [1 \quad 0]X \end{aligned}$ <p>b:</p> $\begin{aligned} X &= [x \quad \dot{x}]^t \\ \dot{X} &= \begin{bmatrix} 0 & 1 \\ -\frac{k}{m} & -\frac{c}{m} \end{bmatrix} X + \begin{bmatrix} 0 \\ \frac{1}{m} \end{bmatrix} F \\ Y &= [1 \quad 0]X \end{aligned}$ <p>b:</p> $\begin{aligned} X &= [x \quad \dot{x}]^t \\ \dot{X} &= \begin{bmatrix} 0 & 1 \\ -\frac{c}{m} & -\frac{k}{m} \end{bmatrix} X + \begin{bmatrix} 0 \\ \frac{1}{m} \end{bmatrix} F \\ Y &= [1 \quad 0]X \end{aligned}$