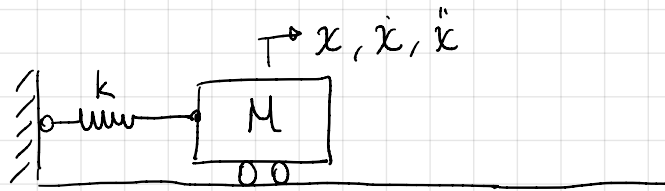



Natural Frequency of SDOF System



→ undamped

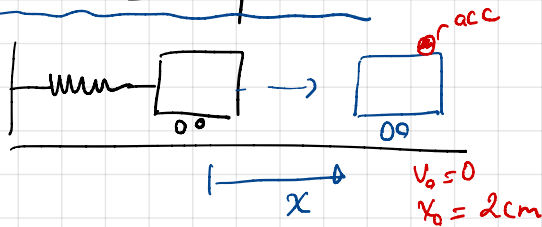
→ damped

→ underdamped ($\zeta < 1$)

→ critically damped ($\zeta = 1$)

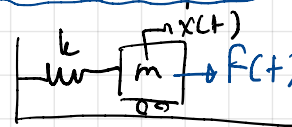
→ overdamped ($\zeta > 1$)

→ Free Response



$$x(t) = A \sin(\omega_n t)$$

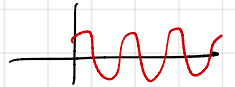
→ Forced Response



step input:

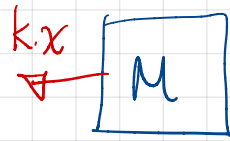
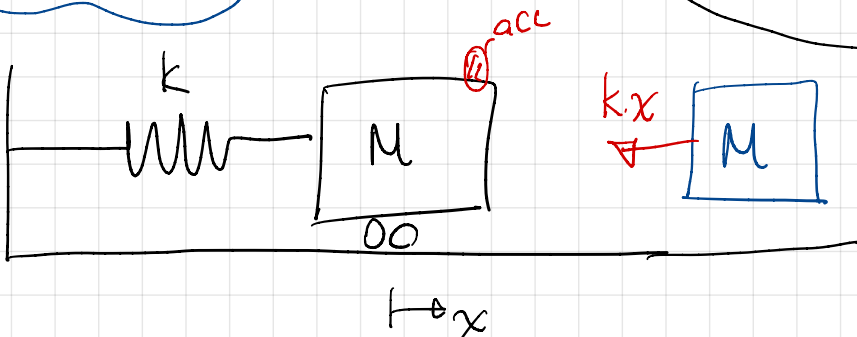
sinusoidal input:

$$F(t) = A \sin(\omega t)$$



ω → forcing freq. excitation freq.

$$x(t) = x_t(t) + x_{ss}(t)$$



$$M = m_c + m_t + m_{twid} + m_{ac}$$

$$\Sigma F = M \cdot \ddot{x} \Rightarrow -kx = M \cdot \ddot{x} \Rightarrow M \ddot{x} + kx = 0$$

The equation of motion: $M \ddot{x}(t) + kx(t) = 0$

$$\ddot{x}(t) + \frac{k}{M} x(t) = 0 \Rightarrow \ddot{x}(t) + \omega_n^2 x(t) = 0$$

$$x(t) = A \sin \omega_n t, \quad \dot{x}(t) = A \omega_n \cos \omega_n t, \quad \ddot{x}(t) = -A \omega_n^2 \sin \omega_n t$$

$$\ddot{x}(t) = -\omega_n^2 x(t)$$

$$\ddot{x}(t) + \frac{k}{M} x(t) = 0 \Rightarrow -\omega_n^2 x(t) + \frac{k}{M} x(t) = 0$$

$$-\omega_n^2 x(t) + \frac{k}{m} x(t) = 0$$

$$\left[-\omega_n^2 + \frac{k}{m} \right] \cdot \underbrace{x(t)}_{x \neq 0} = 0$$

$$-\omega_n^2 = -\frac{k}{m}$$

$$\boxed{\omega_n^2 = \frac{k}{m}}$$

$$\rightarrow \text{EOM: } \ddot{x}(t) + \frac{k}{m} x(t) = 0$$

$$\ddot{x}(t) + \omega_n^2 x(t) = 0$$

$$\omega_n^2 = \frac{k}{m} \Rightarrow \omega_n = \sqrt{\frac{k}{m}}$$

ω_n = Natural Freq. in rad/sec

$$\omega_n = 2\pi f_n$$

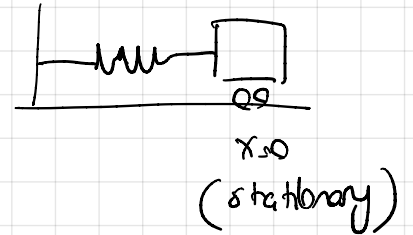
f_n = Natural Freq. in Hz

$$f_n = \frac{1}{T}, \quad T: \text{period (sec)}$$

$$a \cdot b = 0$$

$$\swarrow \quad \searrow$$

$$a = 0 \quad b = 0$$



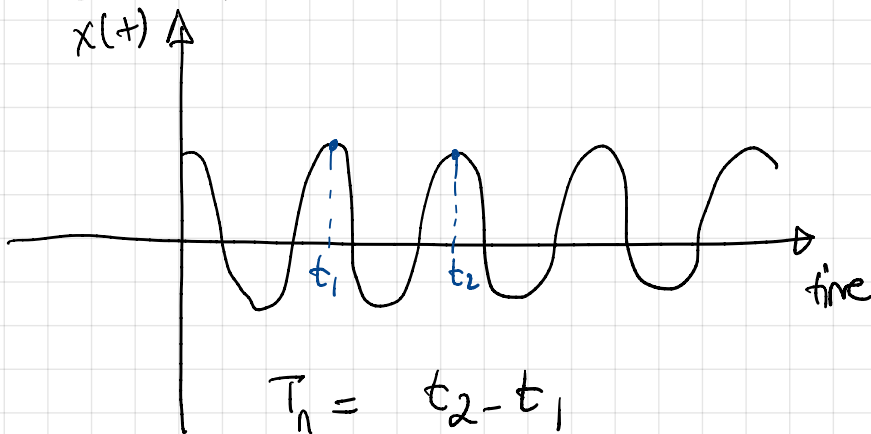
① (*) If you know the total mass and stiffness:

$$\omega_n = \sqrt{\frac{k}{m}}$$

$$f_n = \frac{1}{2\pi} \omega_n$$

② (*) If you have the free response:

undamped

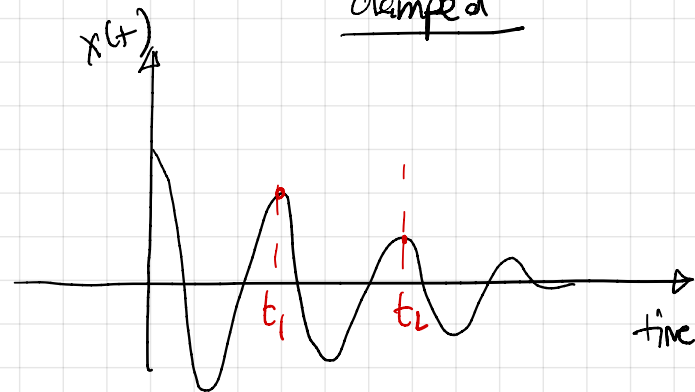


$$T_n = t_2 - t_1$$

$$f_n = \frac{1}{T}$$

$$\omega_n = 2\pi f$$

damped



$$T_d = t_2 - t_1$$

$$f_d = \frac{1}{T}$$

$$\omega_d = 2\pi f$$