

# Dynamic Compensators and Stability Margins

## Control Engineering (Reguleringsteknik)

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The Maersk Mc-Kinney Moller Institute  
University of Southern Denmark

# Agenda



Introduction

Curriculum

Stability Margins

Dynamic Compensation

Lead Compensation

Lag Compensation



Matematiske og **grafiske metoder til syntese af lineære tidsinvariante systemer**:<sup>1</sup>

- ▶ diskret og kontinuert tilstandsbeskrivelse
- ▶ **analyse i tid og frekvens**
- ▶ **stabilitet, reguleringshastighed, følsomhed** og fejl
- ▶ digitale PI, PID, LEAD og LAG regulatorer (serieregulatorer)
- ▶ tilstandsregulering, pole-placement og tilstands-estimering (observer)
- ▶ optimal regulering (least squares) og optimal tilstands-estimation (Kalman-filter)

### Færdigheder:

Efter gennemførelse af kurset kan den succesfulde studerende:

- ▶ **kunne analysere, dimensionere** og implementere såvel **kontinuert** som tidsdiskret **regulering af lineære tidsinvariante** og stokastiske **systemer**

### Kompetencer:

Efter gennemførelse af kurset kan den succesfulde studerende:

- ▶ **anvende** og implementere **klassiske** og moderne **reguleringsteknikker** for at kunne styre og regulere en robot **hurtig og præcist**

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<sup>1</sup> Based on [https://fagbesk.sam.sdu.dk/?fag\\_id=39673](https://fagbesk.sam.sdu.dk/?fag_id=39673)



The twelve lectures of the course are

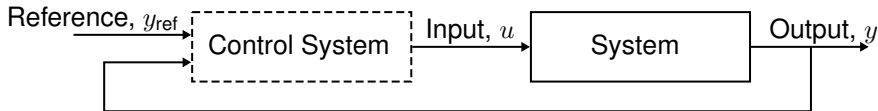
- ▶ **Lecture 1:** Introduction to Linear Time-Invariant Systems
- ▶ **Lecture 2:** Stability and Performance Analysis
- ▶ **Lecture 3:** Introduction to Control
- ▶ **Lecture 4:** Design of PID Controllers
- ▶ **Lecture 5:** Root Locus
- ▶ **Lecture 6:** The Nyquist Plot
- ▶ **Lecture 7:** Dynamic Compensators and Stability Margins
- ▶ **Lecture 8:** Implementation
- ▶ **Lecture 9:** State Feedback
- ▶ **Lecture 10:** Observer Design
- ▶ **Lecture 11:** Optimal Control (Linear Quadratic Control)
- ▶ **Lecture 12:** The Kalman Filter

# Introduction

## Motivating Example



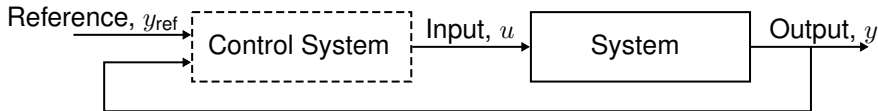
**Task:** Design a cruise control for a car.



- ▶ **Control Input:** Throttle position  $u$
- ▶ **Measured Output:** Velocity of the car  $y$
- ▶ **Reference Input:** Desired velocity of the car  $y_{\text{ref}}$



**Task:** Design a cruise control for a car.

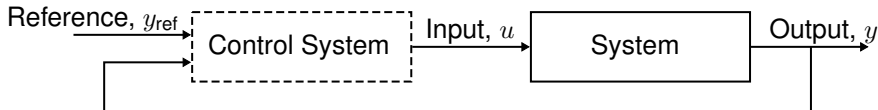


Today, an answer to the following question is provided:

1. How can one ensure that a control is stable despite uncertainties in the system?



**Task:** Design a cruise control for a car.



Today, an answer to the following question is provided:

1. How can one ensure that a control is stable despite uncertainties in the system?
  - ▶ Uncertainties may affect the *gain* of the system (e.g. the inertia).
  - ▶ Uncertainties may affect the *phase* of the system (e.g. communication delays between controller and sensor).

# Stability Margins



Introduction

Curriculum

**Stability Margins**

Dynamic Compensation

Lead Compensation

Lag Compensation



# Stability Margins

## Gain Margin: Definition



The ***gain margin*** is the factor by which the gain can be raised before a system becomes unstable.

# Stability Margins

## Gain Margin: Definition



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Recall that a system is **neutrally stable** for

$$|L(j\omega)| = 1 \quad \text{and} \quad \angle L(j\omega) = 180^\circ$$

where  $L(s) = K(s)G(s)$ .

# Stability Margins

## Gain Margin: Definition

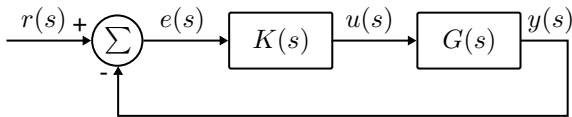


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# Stability Margins

## Phase Margin: Definition



The **phase margin** is the amount by which the phase can be raised before a system becomes unstable (before it exceeds  $-180^\circ$ ).

A system is **neutrally stable** for

$$|L(j\omega)| = 1 \quad \text{and} \quad \angle L(j\omega) = 180^\circ$$

where  $L(s) = K(s)G(s)$ .

# Stability Margins

## Relation to Bode Plot



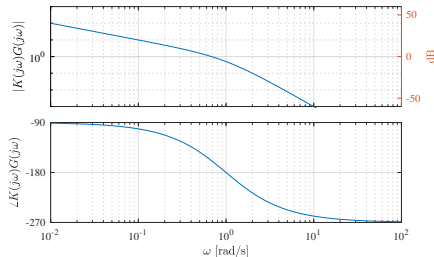
The stability margins can be determined from a Bode plot as (use `margin` in MATLAB)

- **Gain Margin:** Find the value of  $\omega$  where  $\angle K(j\omega)G(j\omega) = -180^\circ$ , and denote it by  $\omega_{GM}$ . The gain margin (in dB) is

$$GM = -|K(j\omega_{GM})G(j\omega_{GM})|$$

- **Phase Margin:** Find the value of  $\omega$  where  $|K(j\omega)G(j\omega)| = 0$  dB, and denote it by  $\omega_c$  (this frequency is called the ***crossover frequency***). The phase margin is

$$PM = \angle K(j\omega_c)G(j\omega_c) + 180^\circ$$



# Stability Margins

## Relation to Bode Plot



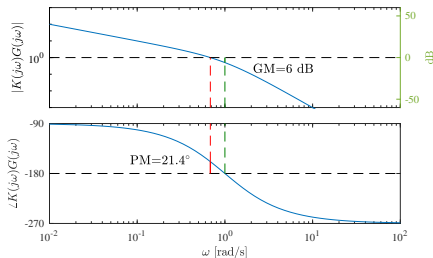
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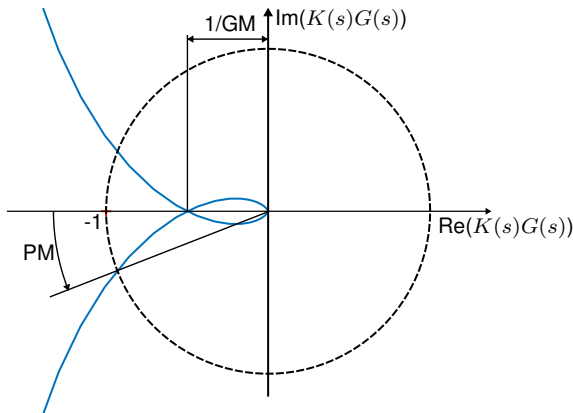


# Stability Margins

Relation to Nyquist Plot



The stability margins can also be determined from a Nyquist plot, where the margins are determined by the closeness of the Nyquist plot to the point -1.



# Stability Margins

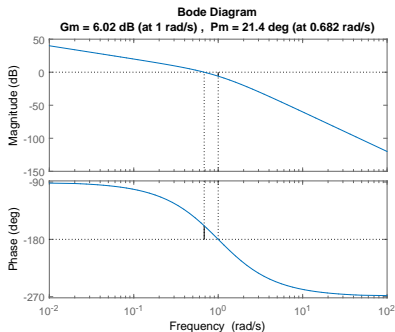
Example: Interpretation of Margins



Consider the loop gain

$$L(s) = K(s)G(s) = \frac{1}{s(s+1)^2}$$

with the following Bode plot.





# Stability Margins

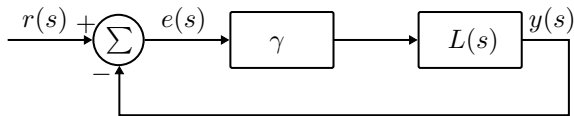
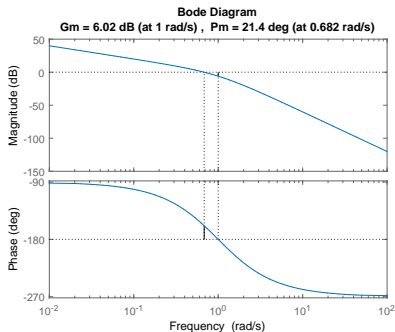
Example: Interpretation of Margins



Consider the loop gain

$$L(s) = K(s)G(s) = \frac{1}{s(s+1)^2}$$

with the following Bode plot.



The gain  $\gamma$  can be increased to 2 before the system becomes unstable.

# Stability Margins

Example: Parameter Change



Consider a loop gain

$$L(s) = \frac{1.3(s + 2)}{s^3 + s^2 + bs + 1}$$

where  $b$  is some parameter.

# Stability Margins

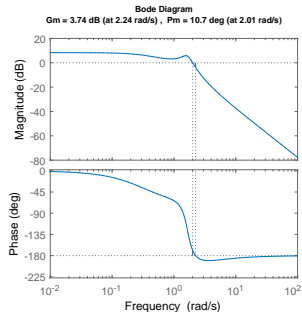
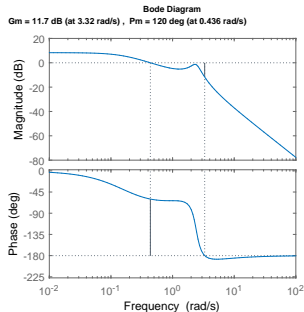
## Example: Parameter Change



Consider a loop gain

$$L(s) = \frac{1.3(s + 2)}{s^3 + s^2 + bs + 1}$$

where  $b$  is some parameter. A parameter change from  $b = 6$  (left) to  $b = 3$  (right) changes the margins significantly.



# Stability Margins

Example: Shortcoming of Phase and Gain Margins



The phase and gain margins determine stability properties, when only the gain OR phase is changed.

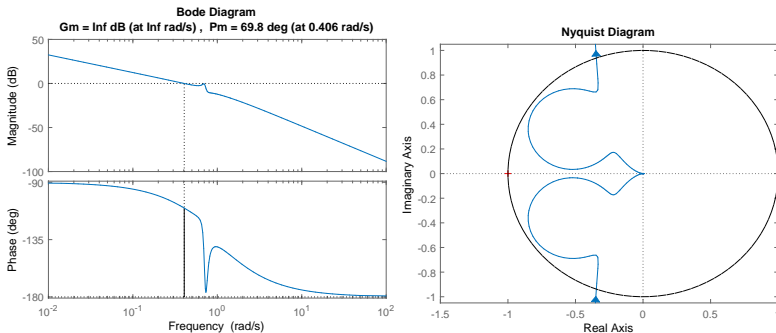
# Stability Margins

Example: Shortcoming of Phase and Gain Margins



The phase and gain margins determine stability properties, when only the gain OR phase is changed.

If the gain and phase is changed simultaneously, issues can occur.



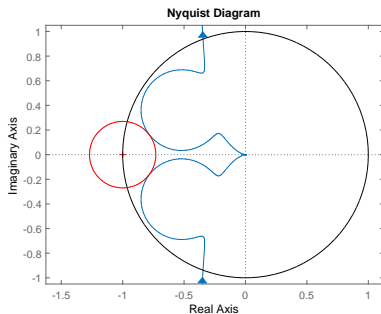
# Stability Margins

Example: Shortcoming of Phase and Gain Margins



The phase and gain margins determine stability properties, when only the gain OR phase is changed.

This is the motivation for looking at the vector margin that gives the shortest distance between the Nyquist plot and -1.



# Dynamic Compensation



Introduction  
Curriculum

Stability Margins

Dynamic Compensation  
Lead Compensation  
Lag Compensation

# Dynamic Compensation

## Overview



Two types of compensators are considered

- ▶ **Lead Compensation:** Approximates the PD control, i.e., it lowers the rise time and decreases the overshoot.
- ▶ **Lag Compensation:** Approximates the PI control, i.e., it improves the steady state tracking.



# Dynamic Compensation

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Both compensators are given by the transfer function

$$D(s) = K \frac{s + z}{s + p}.$$

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Both compensators are given by the transfer function

$$D(s) = K \frac{s + z}{s + p}.$$

- ▶ If  $z < p$ , then  $D(s)$  is called a lead compensation.
- ▶ If  $z > p$ , then  $D(s)$  is called a lag compensation

# Dynamic Compensation

## Lead Compensation: Definition



A lead compensator is given by

$$D(s) = K \frac{s + z}{s + p}.$$

where  $z < p \in \mathbb{R}$ .

# Dynamic Compensation

## Lead Compensation: Definition

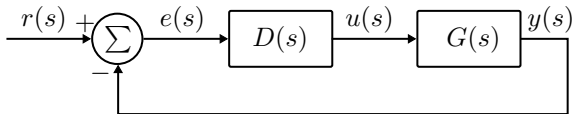


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where  $z < p \in \mathbb{R}$ .

The transfer function  $D(s)$  has a zero followed by a pole, and acts as a filtered PD controller, when used as a feedback controller.



# Dynamic Compensation

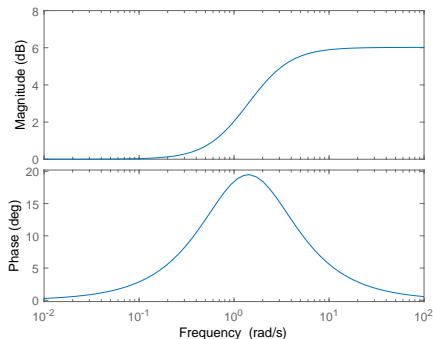
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# Dynamic Compensation

## Lead Compensation: Parameters



A lead compensation is given by

$$D(s) = \frac{Ts + 1}{\alpha Ts + 1} \quad \text{where } \alpha < 1$$

and  $1/\alpha$  is called the **lead ratio**.

# Dynamic Compensation

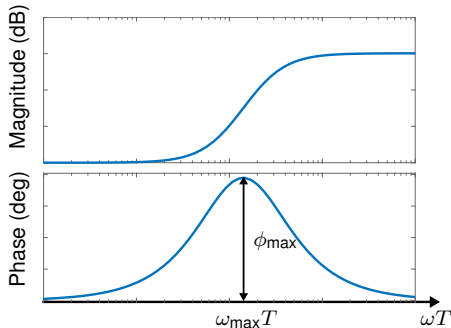
## Lead Compensation: Parameters



A lead compensation is given by

$$D(s) = \frac{Ts + 1}{\alpha Ts + 1} \quad \text{where } \alpha < 1$$

and  $1/\alpha$  is called the **lead ratio**.



We have the following

$$\omega_{\max} = \frac{1}{T\sqrt{\alpha}} = \sqrt{|z||p|}$$

and

$$\sin \phi_{\max} = \frac{1 - \alpha}{1 + \alpha}$$

# Dynamic Compensation

## Lead Compensation: Design Procedure (Bode Plot)



1. Determine open-loop gain  $K$  to satisfy error or bandwidth requirements:
  - 1.1 To meet error requirement, pick  $K$  to satisfy error constants ( $K_p$ ,  $K_V$ ,  $K_a$ ) so that  $e_{ss}$  error specification is met.
  - 1.2 Alternatively, to meet bandwidth requirement, pick  $K$  so that the open-loop crossover frequency is a factor of two below the desired closed-loop bandwidth.
2. Evaluate the phase margin of the uncompensated system using the value  $K$  obtained from Step 1.
3. Allow for extra margin (about  $10^\circ$ ), and determine the needed phase lead  $\phi_{\max}$  (one lead compensation should contribute a maximum of  $60^\circ$  to the phase).
4. Determine  $\alpha$  from

$$\sin \phi_{\max} = \frac{1 - \alpha}{1 + \alpha}$$



# Dynamic Compensation

Lead Compensation: Design Procedure (Bode Plot)



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4. Determine  $\alpha$  from

$$\sin \phi_{\max} = \frac{1 - \alpha}{1 + \alpha}$$

5. Pick  $\omega_{\max}$  to be the crossover frequency; thus, the zero is at  $1/T = \omega_{\max}\sqrt{\alpha}$  and the pole is at  $1/\alpha T = \omega_{\max}/\sqrt{\alpha}$ .
6. Draw the compensated frequency response and check the phase margin.
7. Iterate on the design. Adjust compensator parameters (poles, zeros, and gain) until all specifications are met.

# Dynamic Compensation

## Lead Compensation: Example (1)



Consider the system

$$KG(s) = \frac{K}{(s/0.5 + 1)(s + 1)(s/2 + 1)}$$

# Dynamic Compensation

## Lead Compensation: Example (1)



Consider the system

$$KG(s) = \frac{K}{(s/0.5 + 1)(s + 1)(s/2 + 1)}$$

Design a lead compensator such that the steady state error to a step input is

$$\frac{1}{1 + K_p}$$

where  $K_p = 9$  and the phase margin is at least  $25^\circ$ .

# Dynamic Compensation

## Lead Compensation: Example (2)



Step 1: Meet steady state error requirement.

We use the Final Value Theorem to find  $K$

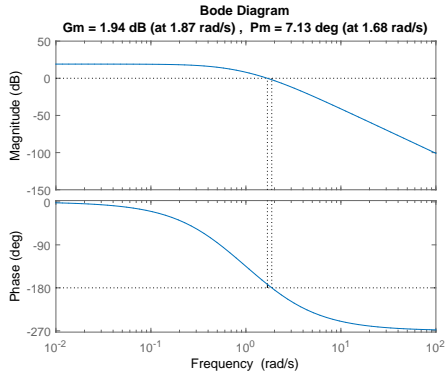
$$K_p = \lim_{s \rightarrow 0} KG(s) = K = 9$$

# Dynamic Compensation

## Lead Compensation: Example (3)



Step 2: The Bode plot of  $KG(s)$  with  $K = 9$  is shown below.



The phase margin is  $7^\circ$  at  $\omega = 1.68$  rad/s.

# Dynamic Compensation

## Lead Compensation: Example (4)



Step 3: To allow a phase margin of  $25^\circ$  (requirement) plus  $10^\circ$  (extra margin), the needed phase lead is

$$\phi_{\max} = 25^\circ + 10^\circ - 7^\circ = 28^\circ$$

# Dynamic Compensation

## Lead Compensation: Example (4)



Step 3: To allow a phase margin of  $25^\circ$  (requirement) plus  $10^\circ$  (extra margin), the needed phase lead is

$$\phi_{\max} = 25^\circ + 10^\circ - 7^\circ = 28^\circ$$

Step 4: The value of  $\alpha$  is computed as

$$\alpha = \frac{1 - \sin \phi_{\max}}{1 + \sin \phi_{\max}} = 0.3610$$

# Dynamic Compensation

## Lead Compensation: Example (4)



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Step 4: The value of  $\alpha$  is computed as

$$\alpha = \frac{1 - \sin \phi_{\max}}{1 + \sin \phi_{\max}} = 0.3610$$

Step 5: Pick  $\omega_{\max}$  to be at the crossover frequency, i.e.,  $\omega_{\max} = 1.68$  rad/s. Thereby,

$$T = \frac{1}{\omega_{\max} \sqrt{\alpha}} = 0.9906$$

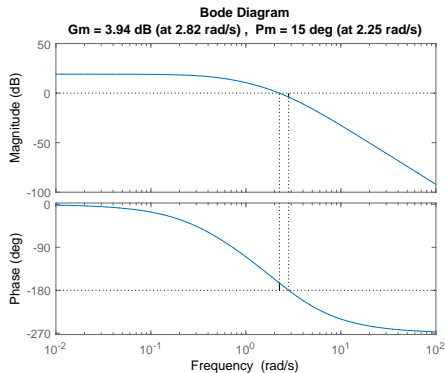


# Dynamic Compensation

## Lead Compensation: Example (5)



Step 6: Verify that the response satisfies the requirements. Since the crossover frequency has changed to 2.25 rad/s, the design does not work.



# Dynamic Compensation

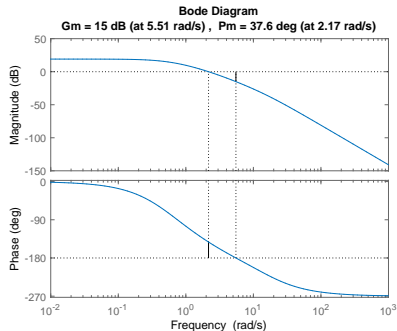
## Lead Compensation: Example (5)



Step 7: The compensator is redesigned, as

$$D(s) = \frac{s/1.5 + 1}{s/15 + 1}$$

This gives the following frequency response



# Dynamic Compensation

## Lag Compensation: Definition



A lag compensator is given by

$$D(s) = K \frac{s + z}{s + p}$$

where  $z > p \in \mathbb{R}$

# Dynamic Compensation

## Lag Compensation: Definition

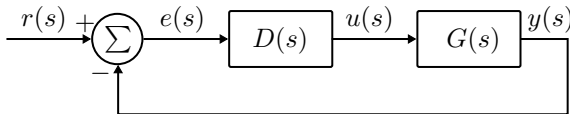


A lag compensator is given by

$$D(s) = K \frac{s + z}{s + p} \quad \text{or} \quad D(s) = K_0 \alpha \frac{Ts + 1}{\alpha Ts + 1}$$

where  $z > p \in \mathbb{R}$  and  $\alpha > 1$ .

The transfer function  $D(s)$  has a pole followed by a zero, and approximates a PI controller, when used as a feedback controller.



# Dynamic Compensation

## Lag Compensation: Definition

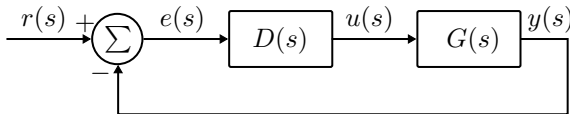


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The transfer function  $D(s)$  has a pole followed by a zero, and approximates a PI controller, when used as a feedback controller.



**Idea:** Improve the steady-state performance without affecting the other dynamics.

# Dynamic Compensation

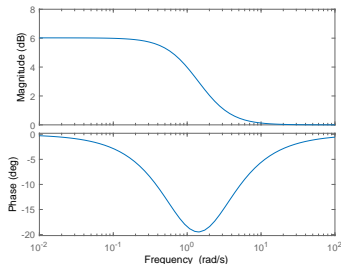
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where  $z > p \in \mathbb{R}$  and  $\alpha > 1$ .



# Dynamic Compensation

## Lag Compensation: Design Procedure (Bode Plot)



1. Determine open-loop gain  $K$  that will meet the phase margin requirement without compensation.
2. Draw the Bode plot of the uncompensated system with crossover frequency from Step 1, and evaluate the low-frequency gain.
3. Determine  $\alpha$  to meet the low-frequency gain error requirement.
4. Choose the corner frequency  $\omega = 1/T$  (the zero of the lag compensator) to be one decade below the crossover frequency  $\omega_c$ .
5. The other corner frequency (the pole location of the lag compensator) is  $\omega = 1/\alpha T$
6. Iterate on the design. Adjust compensator parameters (poles, zeros, and gain) until all specifications are met.

# Dynamic Compensation

## Lag Compensation: Example (1)



Design a lag compensator so that the phase margin is at least  $40^\circ$  and  $K_p = 9$  for the system

$$KG(s) = \frac{K}{(s/0.5 + 1)(s + 1)(s/2 + 1)}$$



# Dynamic Compensation

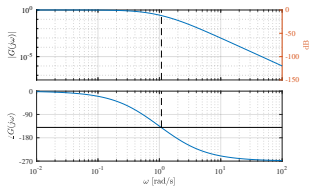
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Step 1: Determine  $K$  such that  $PM > 40^\circ$ .



# Dynamic Compensation

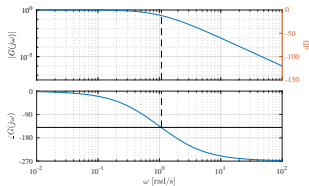
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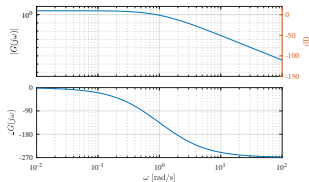
We choose  $K = 3$ .

# Dynamic Compensation

## Lag Compensation: Example (2)



Step 2: Draw Bode plot, and evaluate low-frequency gain of  $KG(s)$ .

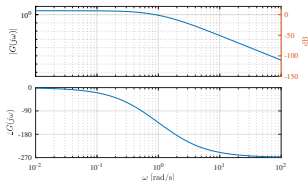


# Dynamic Compensation

## Lag Compensation: Example (2)



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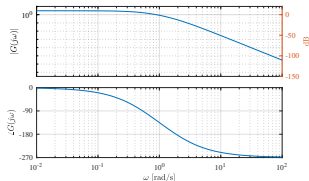
The phase margin is  $53.5^\circ$  and the low-frequency gain is 10 dB (or 3.16 times).

# Dynamic Compensation

## Lag Compensation: Example (2)



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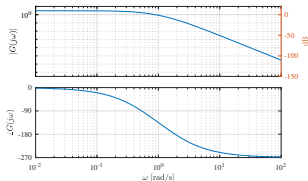
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# Dynamic Compensation

## Lag Compensation: Example (2)



Step 2: Draw Bode plot, and evaluate low-frequency gain of  $KG(s)$ .



The phase margin is  $53.5^\circ$  and the low-frequency gain is 10 dB (or 3.16 times). Step 3: Determine  $\alpha$  such that desired low-frequency gain is attained.

To get  $K_p = 9$ ,  $\alpha$  must be 3.

# Dynamic Compensation

## Lag Compensation: Example (3)



Step 4: Choose the zero of the lag compensator (corner frequency  $\omega = 1/T$ ) to be one octave slower than the crossover frequency, i.e.,  $\omega = 0.2$  rad/s (and  $T = 5$ ).

# Dynamic Compensation

## Lag Compensation: Example (3)



Step 4: Choose the zero of the lag compensator (corner frequency  $\omega = 1/T$ ) to be one octave slower than the crossover frequency, i.e.,  $\omega = 0.2$  rad/s (and  $T = 5$ ).

Step 5: The corner frequency of the pole of the lag compensator should be chosen to be  $\omega = 1/\alpha T$ . Thereby, the lag compensator is given by

$$D(s) = 3 \frac{5s + 1}{15s + 1}$$



# Dynamic Compensation

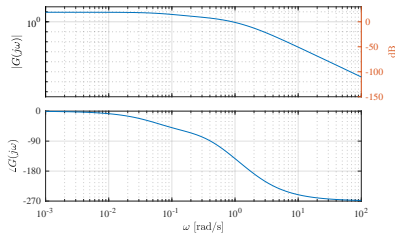
## Lag Compensation: Example (3)



Step 4: Choose the zero of the lag compensator (corner frequency  $\omega = 1/T$ ) to be one octave slower than the crossover frequency, i.e.,  $\omega = 0.2$  rad/s (and  $T = 5$ ).

Step 5: The corner frequency of the pole of the lag compensator should be chosen to be  $\omega = 1/\alpha T$ . Thereby, the lag compensator is given by

$$D(s) = 3 \frac{5s + 1}{15s + 1}$$



The system complies with the specification.