# THERMOFLUIDSTREAM COMMUNITY EVENT PUMPS

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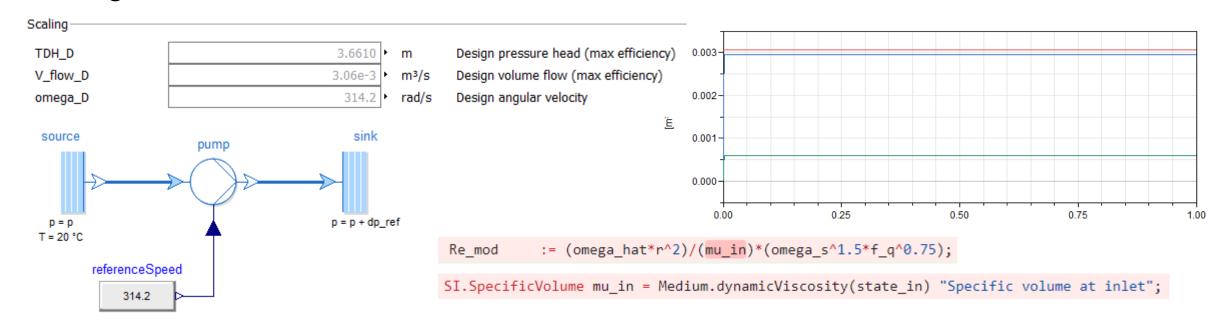
**Aircraft Systems Dynamics – Aircraft Energy Systems** 



### **Motivation – Faulty Pump Model**



 At design angular velocity and design head the volume flow rate was not the design volume flow rate



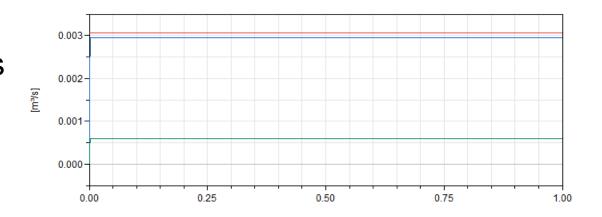
■ Reynolds number (missing density) → fixed 90% of the error but...

### **Motivation – Too complicated**



- Approach to extrapolate measurements data (water) to high viscosity fluids like oil not suitable for water itself since scaling factors are not equal to 1 for the reference fluid (water) → still 5% error
- Implementation as a function → hard to debug

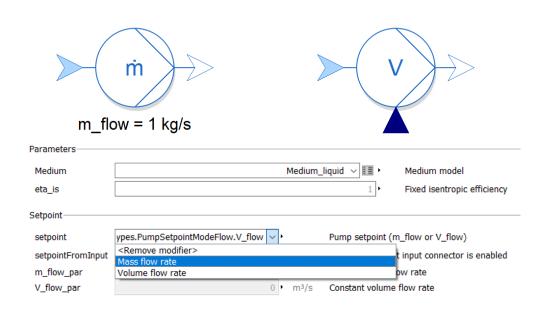
→ Decision to design new pump models

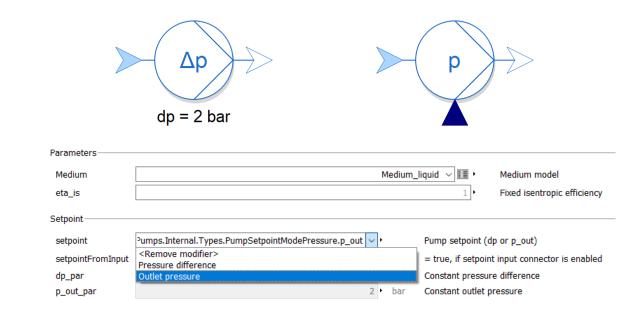


### **SimplePump**



- Ideally controlled
  - Mass flow rate  $\dot{m}$  or volume flow rate  $\dot{V}$  FlowControlledSimplePump
  - Pressure difference  $\Delta p$  or outlet pressure  $p_{\text{out}}$  Pressure Controlled Simple Pump
- Setpoint as paramater or input signal

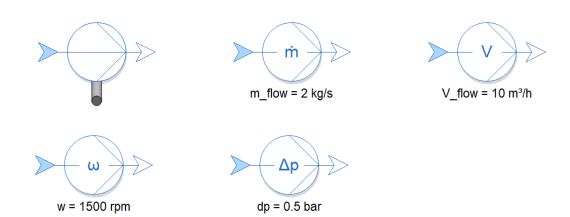




### CentrifugalPump



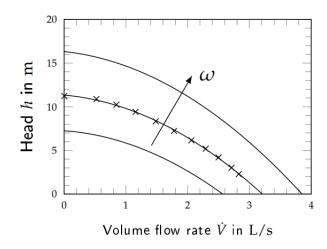
- Ideally controlled
  - Mass flow rate  $\dot{m}$  or volume flow rate  $\dot{V}$  FlowControlledCentrifugalPump
  - Pressure difference  $\Delta p$  or outlet pressure  $p_{\text{out}} Pressure Controlled Centrifugal Pump$
  - Angular velocity ω SpeedControlledCentrifugalPump
- Or mechanical flange connector CentrifugalPump



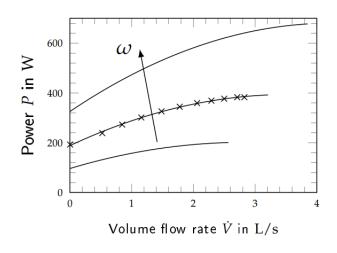
### CentrifugalPump



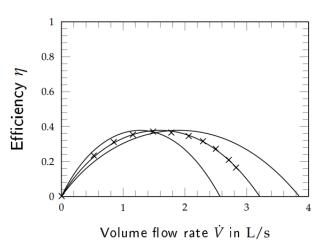
- Approximation of head and power with quadratic polynomial
- Similarity laws for arbitrary speeds ( $\dot{V} \sim \omega$ ,  $h \sim \omega^2$ ,  $P \sim \rho \omega^3$ )
- 6 Coefficients → BasedOnMeasurements or BasedOnCoefficients



$$h = \left(c_1 \left(\frac{\omega}{\omega_{\text{ref}}}\right)^2 + c_2 \frac{\omega}{\omega_{\text{ref}}} \dot{V} + c_3 \dot{V}^2\right)$$



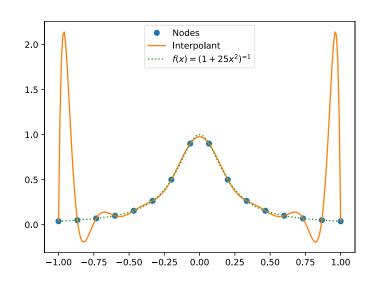
$$h = \left(c_1 \left(\frac{\omega}{\omega_{\text{ref}}}\right)^2 + c_2 \frac{\omega}{\omega_{\text{ref}}} \dot{V} + c_3 \dot{V}^2\right) \qquad P = c_1 \left(\frac{\omega}{\omega_{\text{ref}}}\right)^3 + c_2 \left(\frac{\omega}{\omega_{\text{ref}}}\right)^2 \dot{V} + c_3 \frac{\omega}{\omega_{\text{ref}}} \dot{V}^2$$



# Comparison to Modelica.Fluid



- Approximation instead of Interpolation (no Oscillation, Runge phenomena)
- No function call (works fine for  $\omega = 0$ )



### Modelica.Fluid

$$h = \left(\frac{\omega}{\omega_{\text{ref}}}\right)^{2} \underbrace{\left(c_{1} + c_{2} \frac{\dot{V}}{\omega/\omega_{\text{ref}}} + \ldots + c_{n} \left(\frac{\dot{V}}{\omega/\omega_{\text{ref}}}\right)^{n-1}\right)}_{f_{\text{poly}}\left(\frac{\dot{V}}{\omega/\omega_{\text{opf}}}\right)}$$

$$P = \left(\frac{\omega}{\omega_{\text{ref}}}\right)^{3} \underbrace{\left(c_{1} + c_{2} \frac{\dot{V}}{\omega/\omega_{\text{ref}}} + c_{3} \left(\frac{\dot{V}}{\omega/\omega_{\text{ref}}}\right)^{2}\right)}_{f_{\text{quad}}\left(\frac{\dot{V}}{\omega/\omega_{\text{ref}}}\right)}$$

### ThermofluidStream

$$h = \left(c_1 \left(\frac{\omega}{\omega_{\text{ref}}}\right)^2 + c_2 \frac{\omega}{\omega_{\text{ref}}} \dot{V} + c_3 \dot{V}^2\right)$$

$$P = c_1 \left(\frac{\omega}{\omega_{\text{ref}}}\right)^3 + c_2 \left(\frac{\omega}{\omega_{\text{ref}}}\right)^2 \dot{V} + c_3 \frac{\omega}{\omega_{\text{ref}}} \dot{V}^2$$

# CentrifugalPump BasedOnMeasurements



Parameterized by 3 vectors of measurement data of:

- Volume flow rate  $\{\dot{V}_1, \dots, \dot{V}_n\}$
- Head/pressure difference  $\{h_1, ..., h_n\}$
- Power  $\{P_1, ..., P_n\}$

at reference speed  $\omega_{\mathrm{ref}}$  and reference density  $\varrho_{\mathrm{ref}}$ 

Volume flow rate			
V_flow	{10.924369747884,20.840336134452,31.09243697478,41.00840336124,52.60504201668,65.37815126064,77.1428 <b>III</b> • m³/h Volume flow rate data points		
Head/Pressure rise			
setHead	_ ·		= true, if head data points shall be given (= false, if pressure difference shall be given instead)
head	{10,5,0}	m	Head data points
dp	{1.6821517064,1.66653242326,1.64291843595,1.6	bar	Pressure rise data points
Power			
Р	{1905.29339941,2202.03582759,2548.86483304,2812.07908132,3146.06691483,3435.07911022,3592.75276695,3710.0977453		

# CentrifugalPump BasedOnCoefficients



### Parameterized by

Head at zero volume flow rate

$$h_{\rm ref} = h(\dot{V} = 0, \omega = \omega_{\rm ref})$$

Volume flow rate at zero head

$$\dot{V}_{\rm ref} = \dot{V}(h=0,\omega=\omega_{\rm ref})$$

Initial slope of head curve

$$c_{2,\text{head}} \approx 0.23 \pm 0.3$$

- Max Efficiency  $\eta_{\text{ref}} = \max \eta$
- Volume flow rate at max efficiency  $V/V_{\rm ref}(\max \eta) \approx 0.515 \pm 0.023$
- Curvature of power curve

$$c_{3,\text{power}} \approx -0.28 \pm 0.09$$

