

Required moment at B

$$= \alpha PKU \cos \alpha + 2\pi PV^2 [2bc \sin(\alpha+\beta) \cos(\alpha+\gamma) + a^2 \sin 2\alpha]$$

$$= 4a\pi bV^2 P \cos \alpha + 2\pi PV^2 [2bc \sin(\alpha+\beta) \cos(\alpha+\gamma) + a^2 \sin 2\alpha]$$

Navier-Stokes Equation.

With $P(x, y, z)$ as center and edges of length S_x, S_y, S_z parallel to coordinate axes, we construct an elementary rectangular parallelopiped.

Let us consider the fluid motion is viscous. We assume that the fluid element is moving & its mass is $\rho S_x S_y S_z$. Let the coordinate

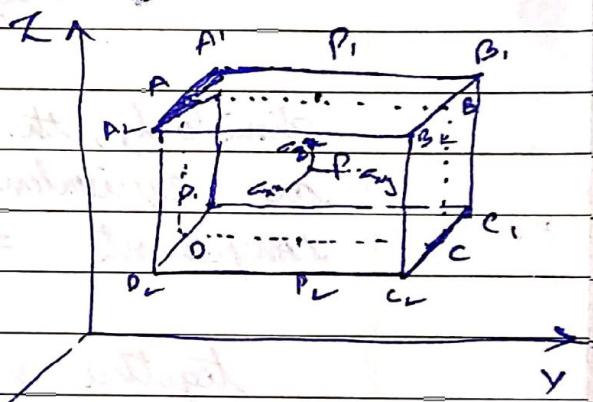
of P_2 & P_1 be $(x + \frac{S_x}{2}, y, z)$ & $(x - \frac{S_x}{2}, y, z)$ respectively. At P the force components // to Ox, Oy, Oz on the rectangular surface ABCD of area $S_y S_z$ through P having \hat{n} as the unit normal are $= (-\sigma_{xx} S_y S_z, -\sigma_{yy} S_y S_z, -\sigma_{zz} S_y S_z)$ — ①

At the point $P_2 (x + S_x/2, y, z)$ the components of the force are $A_2 B_2 C_2 D_2$ is

$$= \left[\left(-\sigma_{xx} + S_y/2 \frac{\partial \sigma_{yy}}{\partial x} \right) S_y S_z, \left(-\sigma_{yy} + S_y/2 \frac{\partial \sigma_{yy}}{\partial x} \right) S_y S_z, \right. \\ \left. \left(-\sigma_{zz} + S_y/2 \frac{\partial \sigma_{zz}}{\partial x} \right) S_y S_z \right] \quad \text{— ②}$$

Only force on P_1 in A, B, C, D,

$$= \left[\mathbf{F} - \left(-\sigma_{yy} - S_y/2 \frac{\partial \sigma_{yy}}{\partial x} \right) S_y S_z, - \left(\sigma_{yy} - S_y/2 \frac{\partial \sigma_{yy}}{\partial x} \right) S_y S_z, \right. \\ \left. - \left(\sigma_{zz} - S_y/2 \frac{\partial \sigma_{zz}}{\partial x} \right) S_y S_z \right] \quad \text{— ③}$$



Hence the forces on II planes $A_2B_2C_2D_2$ & $A_1B_1C_1D_1$, passing through P_2 & P_1 are equivalent to a single force at P with components

$$= \left[\frac{\partial \sigma_{yy}}{\partial n} S_n S_y S_z, \frac{\partial \sigma_{yy}}{\partial n} S_n S_y S_z, \frac{\partial \sigma_{zy}}{\partial n} S_y S_z \right] - (4)$$

together with couple whose moment are

$$\begin{aligned} & -\sigma_{xz} S_n S_y S_z \text{ about } OY \\ & +\sigma_{xy} S_n S_y S_z \text{ about } OZ \end{aligned} \quad \} - (5)$$

Similarly, the forces parallel to the planes ~~each other and~~ \perp to z-axis are equivalent to a single force at P with components $= \left[\frac{\partial \sigma_{zx}}{\partial z}, \frac{\partial \sigma_{zy}}{\partial z}, \frac{\partial \sigma_{zz}}{\partial z} \right] S_n S_y S_z$

together with couple of moment,

$$-\sigma_{zy} S_n S_y S_z \text{ about } OX \& \sigma_{zx} S_n S_y S_z \text{ about } OY.$$

And forces on parallel planes \perp to y-axis are equivalent to single force at P

$$= \left[\frac{\partial \sigma_{yz}}{\partial y}, \frac{\partial \sigma_{yy}}{\partial y}, \frac{\partial \sigma_{zy}}{\partial y} \right] S_n S_y S_z$$

with couple of moment,

$$-\sigma_{yz} S_n S_y S_z \text{ about } OZ \& \sigma_{yy} S_n S_y S_z \text{ about } OX$$

Thus the surface forces on all 6 faces of rectangular II piped are equivalent to a single force at P having components

$$\left(\frac{\partial \sigma_{nn}}{\partial n} + \frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \sigma_{zx}}{\partial z}, \frac{\partial \sigma_{ny}}{\partial n} + \frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \sigma_{zy}}{\partial z}, \frac{\partial \sigma_{nz}}{\partial n} + \frac{\partial \sigma_{zy}}{\partial y} + \frac{\partial \sigma_{zz}}{\partial z} \right)$$

with couple

$$= I (\sigma_{yz} - \sigma_{zy}) S_x S_y S_z, (\sigma_{zx} - \sigma_{xz}) S_x S_y S_z, (\sigma_{xy} - \sigma_{yx}) S_x S_y S_z]$$

Let $\vec{q} = U\hat{i} + V\hat{j} + W\hat{k}$ be the velocity at P &

$\vec{F} = F_x\hat{i} + F_y\hat{j} + F_z\hat{k}$ be the body force per unit mass. The total force is given by

= Body force + surface force

$$= \vec{F} + \vec{S}$$

Total force along i-direction

$$= \left(\frac{\partial \sigma_{xy}}{\partial x} + \frac{\partial \sigma_{yz}}{\partial y} + \frac{\partial \sigma_{zx}}{\partial z} \right) S_x S_y S_z$$

$$+ \rho F_x S_x S_y S_z$$

By Newton's 2nd law :-

$$\rho S_x S_y S_z \frac{DU}{DE} = \left(\frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \sigma_{zz}}{\partial z} + \rho F_x \right) S_x S_y S_z$$

$$\Rightarrow \rho \frac{DU}{DE} = \rho F_x + \frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \sigma_{zz}}{\partial z}$$

$$\text{Also, } \rho \frac{DU}{DE} = \rho F_y + \frac{\partial \sigma_{yy}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \sigma_{zy}}{\partial z}$$

$$\rho \frac{DU}{DE} = \rho F_z + \frac{\partial \sigma_{zz}}{\partial x} + \frac{\partial \sigma_{zy}}{\partial y} + \frac{\partial \sigma_{zz}}{\partial z}$$

From constitutive law of Newtonian Fluid, by
Stoke's law of fluid motion

$$\sigma_{xx} = 2\mu \frac{\partial u}{\partial x} - 2\mu/3 \vec{\nabla} \cdot \vec{q} - p$$

$$\sigma_{yy} = 2\mu \frac{\partial v}{\partial y} - 2\mu/3 \vec{\nabla} \cdot \vec{q} - p$$

$$\sigma_{zz} = 2\mu \frac{\partial w}{\partial z} - 2\mu/3 \vec{\nabla} \cdot \vec{q} - p$$

$$\sigma_{xy} = \sigma_{yz} = \mu \left(\frac{\partial v}{\partial y} + \frac{\partial u}{\partial z} \right)$$

$$\sigma_{yz} = \sigma_{zx} = \mu \left(\frac{\partial w}{\partial z} + \frac{\partial v}{\partial x} \right)$$

$$\sigma_{zx} = \sigma_{xy} = \mu \left(\frac{\partial w}{\partial x} + \frac{\partial u}{\partial y} \right)$$

$$P \frac{DU}{DT} = P F_u + 2\mu \left(\frac{\partial v}{\partial x} \right) + 2\mu/3 \frac{\partial (\vec{\nabla} \cdot \vec{q})}{\partial x} - \frac{\partial P}{\partial x} \\ + \mu \left(\frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 v}{\partial y \partial z} \right) + \mu \left(\frac{\partial^2 w}{\partial z^2} + \frac{\partial^2 u}{\partial z \partial x} \right)$$

$$P \frac{Dq_i}{DT} = P F_u - \frac{\partial P}{\partial x_i} + \frac{\partial}{\partial x_i} \left[\mu \left(2 \frac{\partial v}{\partial x_i} - \frac{4}{3} \vec{\nabla} \cdot \vec{q} \right) \right] \\ + \frac{\partial}{\partial x_j} \left[\mu \left(\frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 v}{\partial y \partial z} \right) \right] + \frac{\partial}{\partial x_k} \left[\mu \left(\frac{\partial^2 w}{\partial z^2} + \frac{\partial^2 u}{\partial z \partial x} \right) \right]$$

$$P \frac{Dv}{DT} = P F_y - \frac{\partial P}{\partial y} \\ - \frac{\partial q_i}{\partial x_i} + \frac{\partial q_j}{\partial x_i}$$

$$P \frac{Dw}{DT}$$

DISSIPATION OF ENERGY

If it is that energy which is dissipated in a viscous fluid in motion on account of internal friction

To determine the rate of dissipation of energy of a fluid due to dissipation :-

Suppose we follow a particle of a viscous incomp fluid of density ρ & volume sv , such that its mass is ρsv . It moves with velocity \vec{q} at any time t . Then KE $T = \frac{1}{2} \rho sv \vec{q}^2$. Hence the rate of change of energy as the particle moves with time is given by,

$$\begin{aligned} \frac{D}{Dt} \left(\frac{1}{2} \rho sv \vec{q}^2 \right) &= \frac{1}{2} \cancel{\frac{D}{Dt} (\rho sv)} \vec{q}^2 \\ &\quad + \rho sv \frac{D}{Dt} (\vec{q}^2) \\ &= \rho sv \vec{q} \cdot \frac{D\vec{q}}{Dt} - \textcircled{1} \end{aligned}$$

Let total volume of the fluid be V & S be the surface area.

$$\begin{aligned} \frac{DT}{DE} &= \frac{d}{dt} \int \frac{1}{2} \rho \vec{q}^2 dv \\ &= \rho \int \vec{q} \cdot \frac{d\vec{q}}{dt} dv - \textcircled{2} \end{aligned}$$

From Navier-Stokes Equation for incompressible fluid.

$$\frac{d\vec{q}}{dt} = \vec{F} - \frac{\nabla P}{\rho} + \nu \vec{\nabla}^2 \vec{q}, \quad \nu = \frac{\mu}{\rho} \text{ of viscosity}$$

$$-\textcircled{3}$$

From ②, ③ :

$$\frac{dT}{dt} = P \int \vec{q} [\vec{F} - \frac{\nabla P}{P} + \nu \vec{\nabla}^2 \vec{q}] dV$$

$$\begin{aligned} \frac{dT}{dt} &= \int \vec{q} (\vec{P} \vec{F}) dV - \cancel{\int \vec{q} \cdot \frac{\nabla P}{P} dV} + \int \vec{q} P \nu \vec{\nabla}^2 \vec{q} dV \\ &= - \left[\int_V \vec{q} \cdot \vec{P} dV + \int_S P \vec{q} \cdot \vec{n} ds \right] - \int_S P \vec{q} \cdot \vec{n} ds \quad \text{--- (4)} \end{aligned}$$

The dissipation first term on RHS of ④ represents rate at which the external force \vec{F} is doing work throughout the mass of the fluid while the second term represents the rate at which pressure is doing work at the boundary. For ideal fluid, the work done by the force \vec{F} in the volume V and work by the pressure at the boundary are same.

$$\frac{dT}{dt} = D = P \int \frac{\mu}{P} \vec{q} \cdot \vec{\nabla}^2 \vec{q} dV = \int \mu \cdot \vec{q} \cdot \vec{\nabla}^2 \vec{q} dV,$$

where D is the dissipation of energy. Now if the flow is rotational s.t. $\vec{\Omega}$ represents the vorticity. Then $\vec{\nabla} \times \vec{\Omega} = \vec{\omega}$
 We know $\vec{\nabla} \times (\vec{\nabla} \times \vec{q}) = \vec{\nabla} (\vec{\nabla} \cdot \vec{q}) - \vec{\nabla}^2 \vec{q} \stackrel{\downarrow 0}{=} -\vec{\nabla}^2 \vec{q}$
 $\Rightarrow \vec{\nabla} \times \vec{\Omega} = -\vec{\nabla}^2 \vec{q}$

$$\Rightarrow \vec{q} \cdot (\vec{\nabla}^2 \vec{q}) = -\vec{q} \cdot (\vec{\nabla} \times \vec{\Omega})$$

Again,

$$\begin{aligned} \vec{\nabla} \cdot (\vec{q} \times \vec{\Omega}) &= \vec{\Omega} \cdot (\vec{q} \times \vec{q}) - \vec{q} \cdot (\vec{\nabla} \times \vec{\Omega}) \\ &= \vec{\Omega} \cdot \vec{\Omega} - \vec{q} \cdot (\vec{\nabla} \times \vec{q}) \end{aligned}$$

$$\Rightarrow \vec{\nabla} \cdot (\vec{q} \times \vec{\Omega}) = |\vec{\Omega}|^2 - \vec{q} \cdot (\vec{\nabla} \times \vec{q})$$

$$\Rightarrow -\vec{q} \cdot (\vec{\nabla} \times \vec{\Omega}) = \vec{\nabla} \cdot (\vec{q} \times \vec{\Omega}) - |\vec{\Omega}|^2$$

This given,

$$\vec{q}(\nabla^2 \vec{q}) = \vec{\nabla}(\vec{q} \times \vec{\omega}) - |\vec{\omega}|^2$$

This gives?

$$D = \int \mu [\vec{\nabla} \cdot (\vec{q} \times \vec{\omega}) - |\vec{\omega}|^2] dV$$

$$= \int \vec{\nabla}(\vec{q} \cdot \vec{\omega})$$

$$= \int_S (\vec{q} \times \vec{\omega}) \cdot \vec{n} ds - \int \mu |\vec{\omega}|^2 dV$$

(no flux at boundary)

If we assume no-slip condition, i.e. $\vec{q} \cdot \vec{n} = 0$, then first term will vanish,

$$D = - \int \mu |\vec{\omega}|^2 dV$$

$$|\vec{\omega}| = \sqrt{\int ((\xi^L + \eta^L + \zeta^L)^2) dV}$$

$$\begin{aligned} \vec{\omega} &= \begin{pmatrix} i & j & k \\ \partial/\partial x & \partial/\partial y & \partial/\partial z \\ u & v & w \end{pmatrix} \\ &= \xi i + \eta j + \zeta k \\ |\vec{\omega}| &= \sqrt{\xi^2 + \eta^2 + \zeta^2} \end{aligned}$$