



Fig. 181 The optimized unstiffened equivalent ellipsoidal shell with the thick apex with $t(\text{apex}) = 0.4$ inch; $W_{\text{imp}}=0.2$ inch; the optimum design is listed in Table 78. State of the shell at load set B (PB) step no. 28. (See Fig. 180). Load set B consists of a number of concentrated inward directed normal **loads** applied along the junction of Shell segments 3 and 4 (Figs. 2, 169, 190, 191) distributed as cos(theta) from theta = 0 to 90 degrees in the circumferential coordinate along Row no. 5 in Shell Units 11 and 12. (See Table a40, except the input datum LT is +1 instead of -1). This load distribution is used because it generates a dent that **locally** resembles the negative of the deformation in Fig. 179, that is, the linear buckling modal imperfection with $n = 1$ circumferential wave.