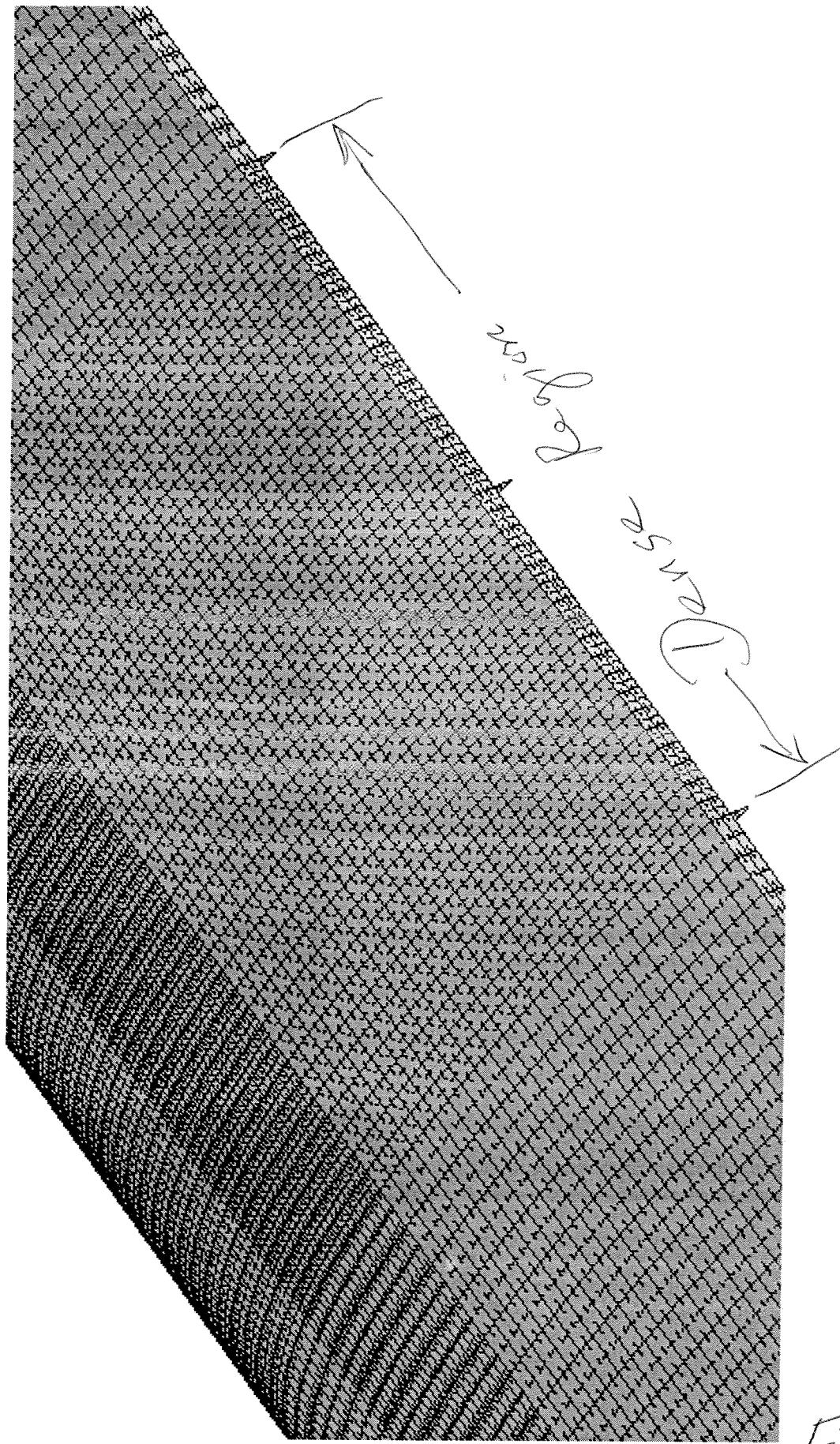


Same as previous fig (zoomed)



# Table 64 Abridged output from 3 STAGS

nasaortholin1.out2.nonuniform.all

OUTPUT FROM STAGS FOR LINEAR BUCKLING RUNS  
IN SEARCH OF THE GENERAL BUCKLING MODE AND  
LOAD FACTOR: ILIN = 1. STAGS 60-DEGREE x 6-RING-BAY MODEL  
WITH NONUNIFORM MESH

Linear buckling runs to  
find general buckling  
mode.

1 nasaortho.out2 (abridged, model with nonuniform mesh)  
shift = 2.608979, 750 negative roots

CONVERGENCE HAS BEEN OBTAINED FOR EIGENVALUES 1 THROUGH 8  
CRITICAL LOAD FACTOR COMBINATION

NO.	EIGENVALUE	LOAD SYSTEM A	LOAD SYSTEM B	@DOF	root
1	2.603670E+00	2.603670E+00	0.000000E+00	220761	746
2	2.606622E+00	2.606622E+00	0.000000E+00	151521	747
3	2.607646E+00	2.607646E+00	0.000000E+00	228465	748
4	2.608357E+00	2.608357E+00	0.000000E+00	215313	749
5	2.608586E+00	2.608586E+00	0.000000E+00	197169	750
6	2.609078E+00	2.609078E+00	0.000000E+00	232833	751
7	2.615415E+00	2.615415E+00	0.000000E+00	122505	752
8	2.615788E+00	2.615788E+00	0.000000E+00	230721	753

2 nasaortho.out2 (abridged, model with nonuniform mesh)  
shift = 2.598, 740 negative roots

CONVERGENCE HAS BEEN OBTAINED FOR EIGENVALUES 1 THROUGH 8  
CRITICAL LOAD FACTOR COMBINATION

NO.	EIGENVALUE	LOAD SYSTEM A	LOAD SYSTEM B	@DOF	root
1	2.595030E+00	2.595030E+00	0.000000E+00	115953	737
2	2.595705E+00	2.595705E+00	0.000000E+00	136833	738
3	2.596621E+00	2.596621E+00	0.000000E+00	136737	739
4	2.597070E+00	2.597070E+00	0.000000E+00	5757	740
5	2.598428E+00	2.598428E+00	0.000000E+00	137145	741
6	2.599055E+00	2.599055E+00	0.000000E+00	215385	742
7	2.599861E+00	2.599861E+00	0.000000E+00	116265	743
8	2.600248E+00	2.600248E+00	0.000000E+00	135345	744

3 nasaortho.out2 (abridged, model with nonuniform mesh)  
shift = 2.600, 743 negative roots (go back to find root 745)

CONVERGENCE HAS BEEN OBTAINED FOR EIGENVALUES 1 THROUGH 8  
CRITICAL LOAD FACTOR COMBINATION

NO.	EIGENVALUE	LOAD SYSTEM A	LOAD SYSTEM B	@DOF	root
1	2.596621E+00	2.596621E+00	0.000000E+00	136737	739
2	2.597070E+00	2.597070E+00	0.000000E+00	5853	740
3	2.598428E+00	2.598428E+00	0.000000E+00	139497	741
4	2.599055E+00	2.599055E+00	0.000000E+00	214377	742
5	2.599861E+00	2.599861E+00	0.000000E+00	116265	743
6	2.600248E+00	2.600248E+00	0.000000E+00	214377	744
7	2.601234E+00	2.601234E+00	0.000000E+00	255697	745 <--general buckling
8	2.603670E+00	2.603670E+00	0.000000E+00	128961	746

STAGS Model with the non-uniform mesh

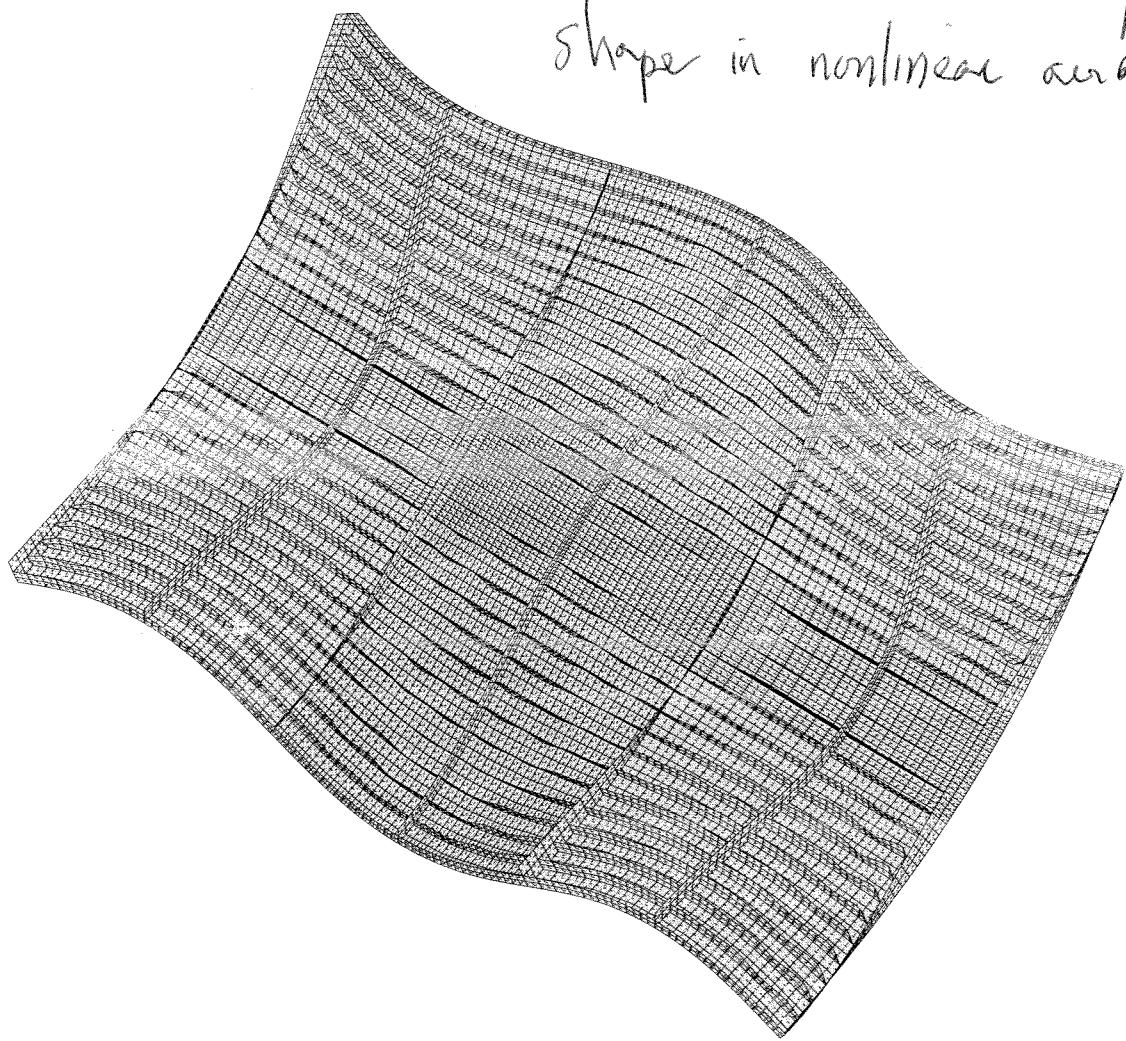
## Table 65 nasaortho, bin

```
nasaortho STAGS INPUT FOR STIFFENED CYL. (STAGSUNIT=SHELL UNITS)
1, $ INDIC=1 is bifur.buckling; INDIC=3 is nonlinear BEGIN B-1
1, $ IPOST=1 means save displacements every IPOSTth step
0, $ ILIST =0 means normal batch-oriented output
0, $ ICOR =0 means projection in; 1 means not in.
1, $ IMPTHE=index for imperfection theory.
0, $ ICHIST=index for crack archive option
0, $ IFLU =0 means no fluid interaction.
-1 $ ISOLVR= 0 means original solver; -1 new solver. END B-1 rec
1.000E+00, $ STLD(1) = starting load factor, System A. BEGIN C-1 rec.
0.000E+00, $ STEP(1) = load factor increment, System A
1.000E+00, $ FACM(1) = maximum load factor, System A
0.000E+00, $ STLD(2) = starting load factor, System B
0.000E+00, $ STEP(2) = load factor increment, System B
0.000E+00, $ FACM(2) = maximum load factor, System B
0 $ ITEMPL=0 means no thermal loads. END C-1 rec.
10000, $ NSEC= number of CPU seconds before run termination
0., $ DELEV is eigenvalue error tolerance (0=.00001)
0 $ IPRINT=0 means print modes, iteration data, END D-2 rec.
1, $ NEIGS= number of eigenvalues sought. BEGIN D-3 rec.
2.601934, $ SHIFT=initial eigenvalue shift
0.000E+00, $ EIGA =lower bound of eigenvalue range
0.000E+00 $ EIGB =upper bound of eigenvalue range. END D-3 rec.
```

See bottom of Table 64

We start over now that we know  
exactly what the eigenvalue shift should be.

General Buckling Mode, to be  
used as an initial imperfection  
shape in nonlinear analysis.



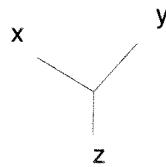
solution scale = 0.4205E+01

mode 1, lpcr = 0.26019E+01

step 0 eigenvector deformed geometry  
linear buckling of perfect shell from STAGS

$\Theta_x -35.84$   
 $\Theta_y -179.86$   
 $\Theta_z 35.63$

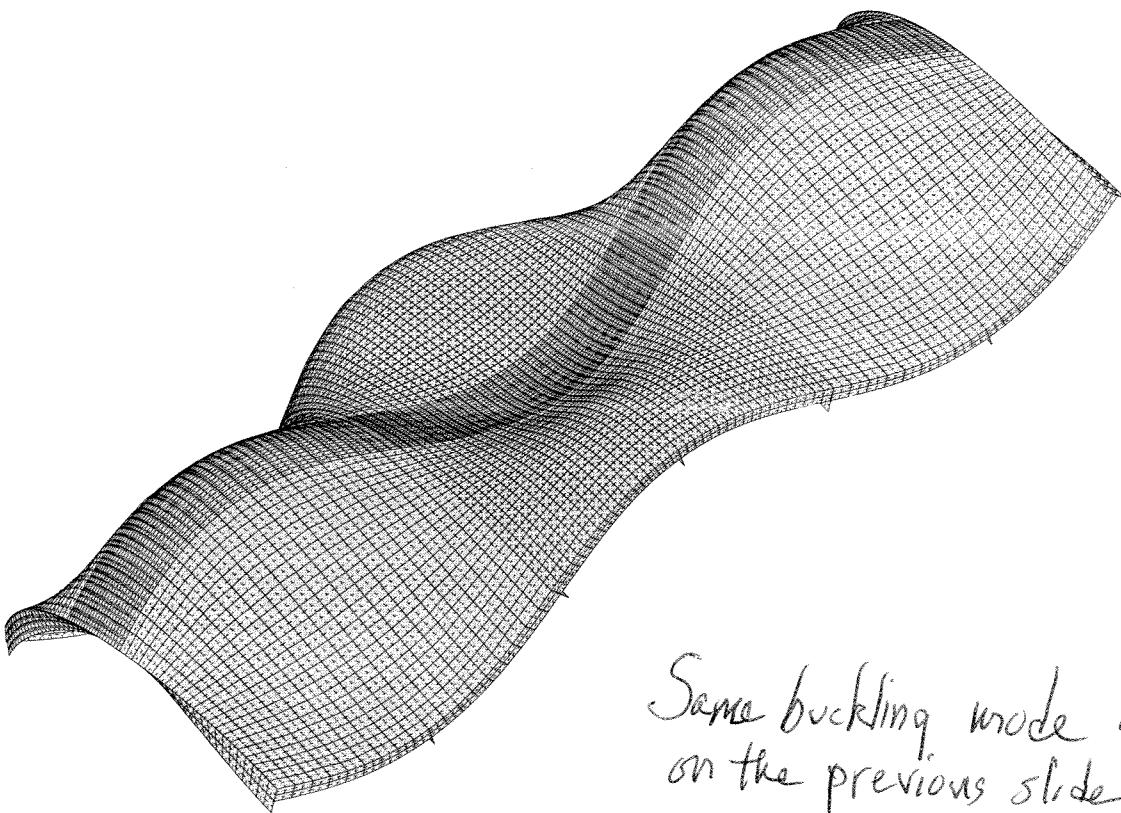
1.336E+01



179

Fig. 66

Compare with Fig. 74



Same buckling mode as  
on the previous slide

solution scale = 0.4521E+01

mode 1, pcr = 0.26019E+01

step 0 eigenvector deformed geometry  
linear buckling of perfect shell from STAGS

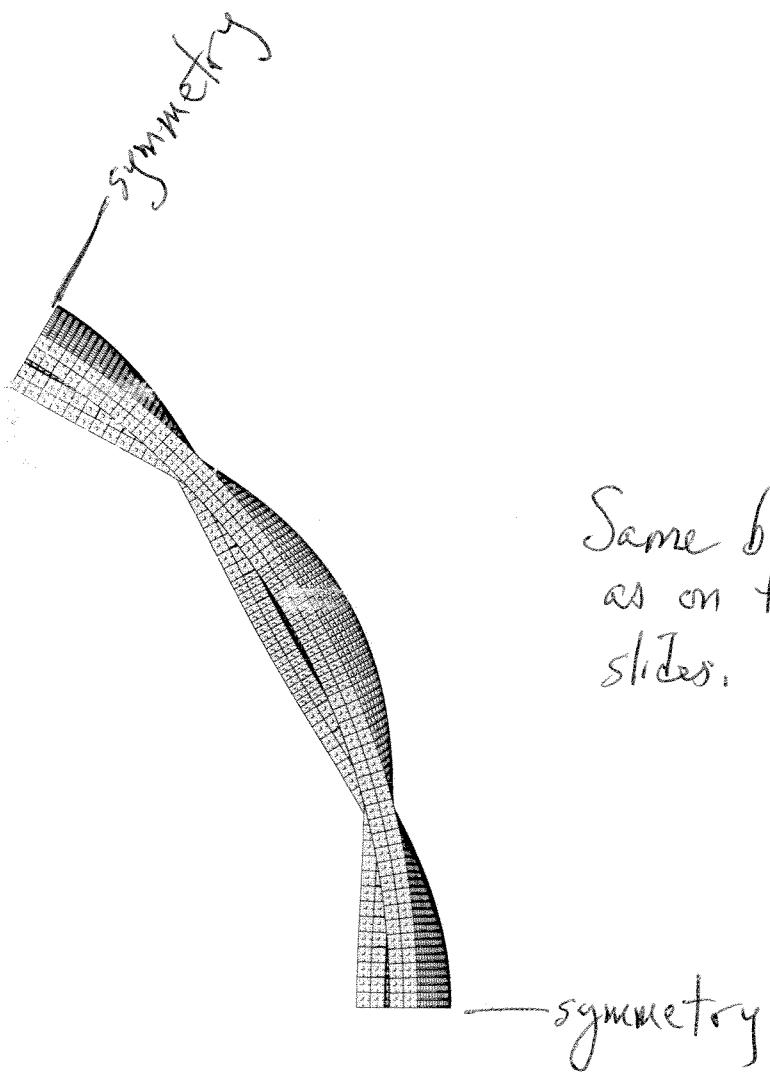
$\Theta_x -35.84$   
 $\Theta_y -13.14$   
 $\Theta_z 35.63$



1.216E+01

180

Fig. 67



Same buckling mode  
as on the previous two  
slides.

solution scale = 0.2082E+01  
mode 1, [pcr = 0.26019E+01]  
step 0 eigenvector deformed geometry  
linear buckling of perfect shell from STAGS

$\Theta_x$  0.00  
 $\Theta_y$  90.00  
 $\Theta_z$  0.00



1.146E+01

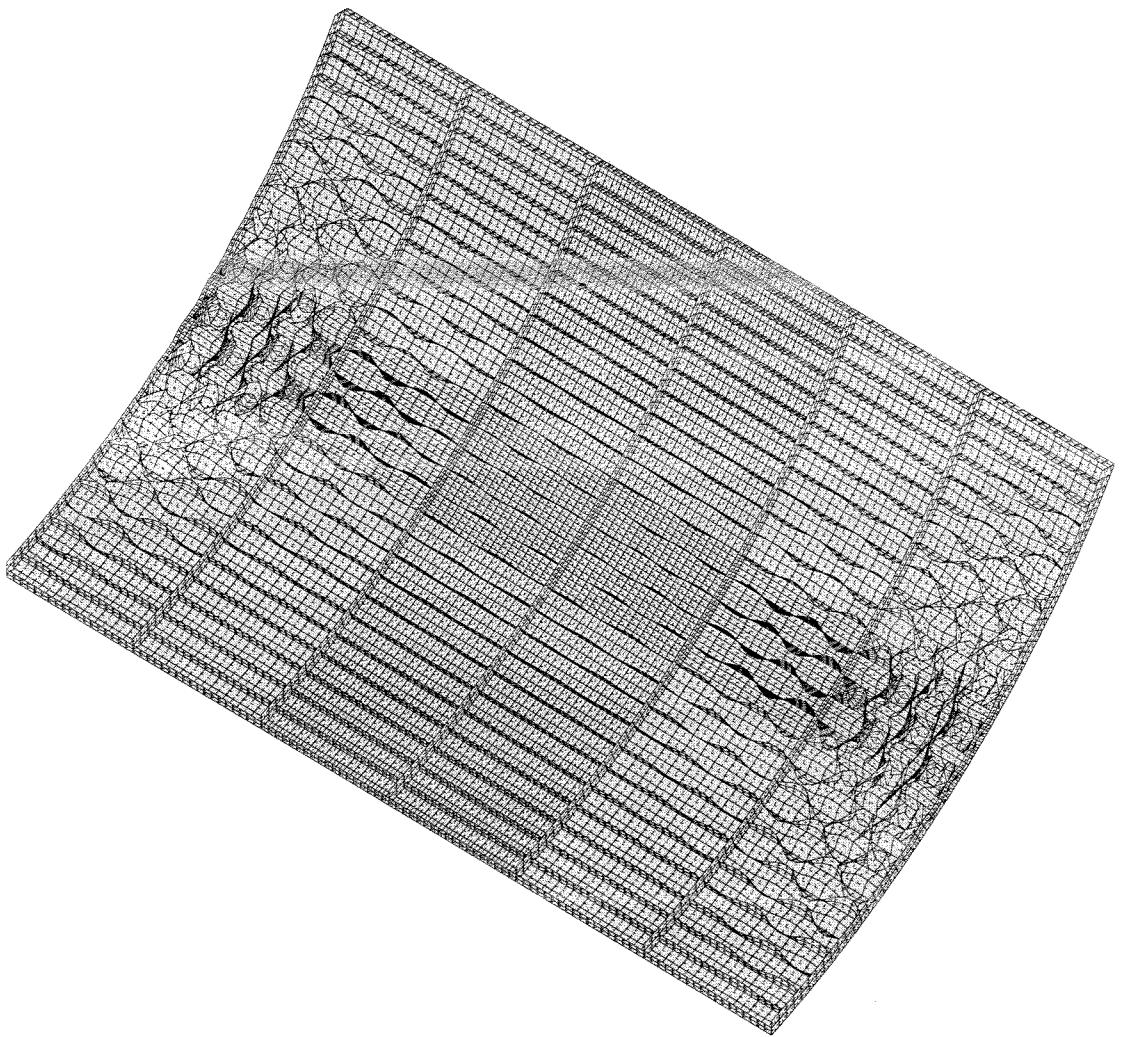
181

Fig. 68

# Table 66 Nasartha, STG (Input for STAGSUNIT)

n 1 \$ Do you want a tutorial session and tutorial output?  
 0 \$ Choose type of STAGS analysis (1,3,4,5,6), INDIC  
 1.000000 0 \$ Restart from ISTARTth load step (0=1st nonlinear soln), ISTART  
 Y 0 \$ Local buckling load factor from PANDA2, EIGLOC  
 note - Are the dimensions in this case in inches?  
 0 \$ Nonlinear (0) or linear (1) kinematic relations?, ILIN  
 68.75000 0 \$ Type 1 for closed (360-deg) cyl. shell, 0 otherwise, ITOTAL  
 50.26548 \$ X-direction length of the STAGS model of the panel: XSTAGS  
 n 22.91667 \$ Panel length in the plane of the screen, L2  
 37 \$ Is the nodal point spacing uniform along the stringer axis?  
 Y 45.83333 \$ Axial callout X(i) where the nodal point spacing changes, X( 1)  
 73 \$ Number of nodes n(i) from X(i-1) to X(i) (n=odd!), n( 1)  
 n 37 \$ Are there any more interior axial stations x where dx changes?  
 -2219 \$ Axial callout X(i) where the nodal point spacing changes, X( 2)  
 0 \$ Number of nodes n(i) from X(i-1) to X(i) (n=odd!), n( 2)  
 n 37 \$ Are there any more interior axial stations x where dx changes?  
 0 \$ Number of nodes n(i) from last X to x = XSTAGS, n( 3)  
 0 \$ Resultant (e.g. lb/in) normal to the plane of screen, Nx  
 0 \$ Resultant (e.g. lb/in) in the plane of the screen, Ny  
 0 \$ In-plane shear in load set A, Nxy  
 0 \$ Normal pressure in STAGS model in Load Set A, p  
 0 \$ Resultant (e.g. lb/in) normal to the plane of screen, Nx0  
 0 \$ Resultant (e.g. lb/in) in the plane of the screen, Ny0  
 0 \$ Normal pressure in STAGS model in Load Set B, p0  
 1 \$ Starting load factor for Load System A, STLD(1)  
 0 \$ Load factor increment for Load System A, STEP(1)  
 1 \$ Maximum load factor for Load System A, FACM(1)  
 0 \$ Starting load factor for Load System B, STLD(2)  
 0 \$ Load factor increment for Load System B, STEP(2)  
 0 \$ Maximum load factor for Load System B, FACM(2)  
 8 \$ How many eigenvalues do you want? NEIGS  
 480 \$ Choose element type (410 or 411 or 480) for panel skin  
 n 162 \$ Have you obtained buckling modes from STAGS for this case?  
 162 \$ Number of stringers in STAGS model of 360-deg. cylinder  
 7 \$ Number of rings in the STAGS model of the panel  
 Y 0 \$ Are there rings at the ends of the panel?  
 100 \$ Number of finite elements between adjacent stringers  
 9 \$ Number of finite elements over circumference, NELCIR  
 3 \$ Stringer model: 1 or 2 or 3 or 4 or 5 (Type H(elp))  
 3 \$ Ring model: 1 or 2 or 3 or 4 or 5 (Type H(elp))  
 -1 \$ Reference surface of cyl: 1=outer, 0=middle, -1=inner  
 n 45 \$ Do you want to use fasteners (they are like rigid links)?  
 n 45 \$ Are the stringers to be "smeared out"?  
 n 45 \$ Is the nodal point spacing uniform around the circumference?  
 24.44444 45 \$ Circ. callout Y(i) where the nodal point spacing changes, Y( 1)  
 Y 35.55555 41 \$ Number of nodes n(i) from Y(i-1) to Y(i) (n=odd!), n( 1)  
 n 45 \$ Are there any more interior axial stations y where dy changes?  
 n 45 \$ Circ. callout Y(i) where the nodal point spacing changes, Y( 2)  
 n 45 \$ Number of nodes n(i) from Y(i-1) to Y(i) (n=odd!), n( 2)  
 n 5 \$ Are there any more interior axial stations y where dy changes?  
 n 5 \$ Number of nodes n(i) from last Y to y = YSTAGS, n( 3)  
 n 5 \$ Are the rings to be "smeared out"?  
 n 3 \$ Number of nodes over height of stiffener webs, NODWEB  
 n 3 \$ Number of nodes over width of stringer flange, NDFLGS  
 n 3 \$ Number of nodes over width of ring flange, NDFLGR  
 Y 13 \$ Do you want stringer(s) with a high nodal point density?  
 Y 14 \$ Number of a stringer to have high mesh density, NUMSTR  
 Y 14 \$ Do you want to choose another stringer?  
 Y 15 \$ Number of a stringer to have high mesh density, NUMSTR  
 Y 15 \$ Do you want to choose another stringer?  
 Y 16 \$ Number of a stringer to have high mesh density, NUMSTR  
 n 16 \$ Do you want to choose another stringer?  
 n 16 \$ Do you want ring(s) with a high nodal point density?  
 n 16 \$ Is there plasticity in this STAGS model?  
 Y 16 \$ Do you want to use the "least-squares" model for torque?  
 Y 16 \$ Is stiffener sidesway permitted at the panel edges?  
 Y 16 \$ Do you want symmetry conditions along the straight edges?

ILIN=0 must be used for local buckling  
and for nonlinear analysis.



solution scale = 0.4627E+01

mode 1, pcr = 0.13506E+01

step 0 eigenvector deformed geometry  
linear buckling of perfect shell from STAGS

$\Theta_x -35.84$   
 $\Theta_y -179.86$   
 $\Theta_z 35.63$

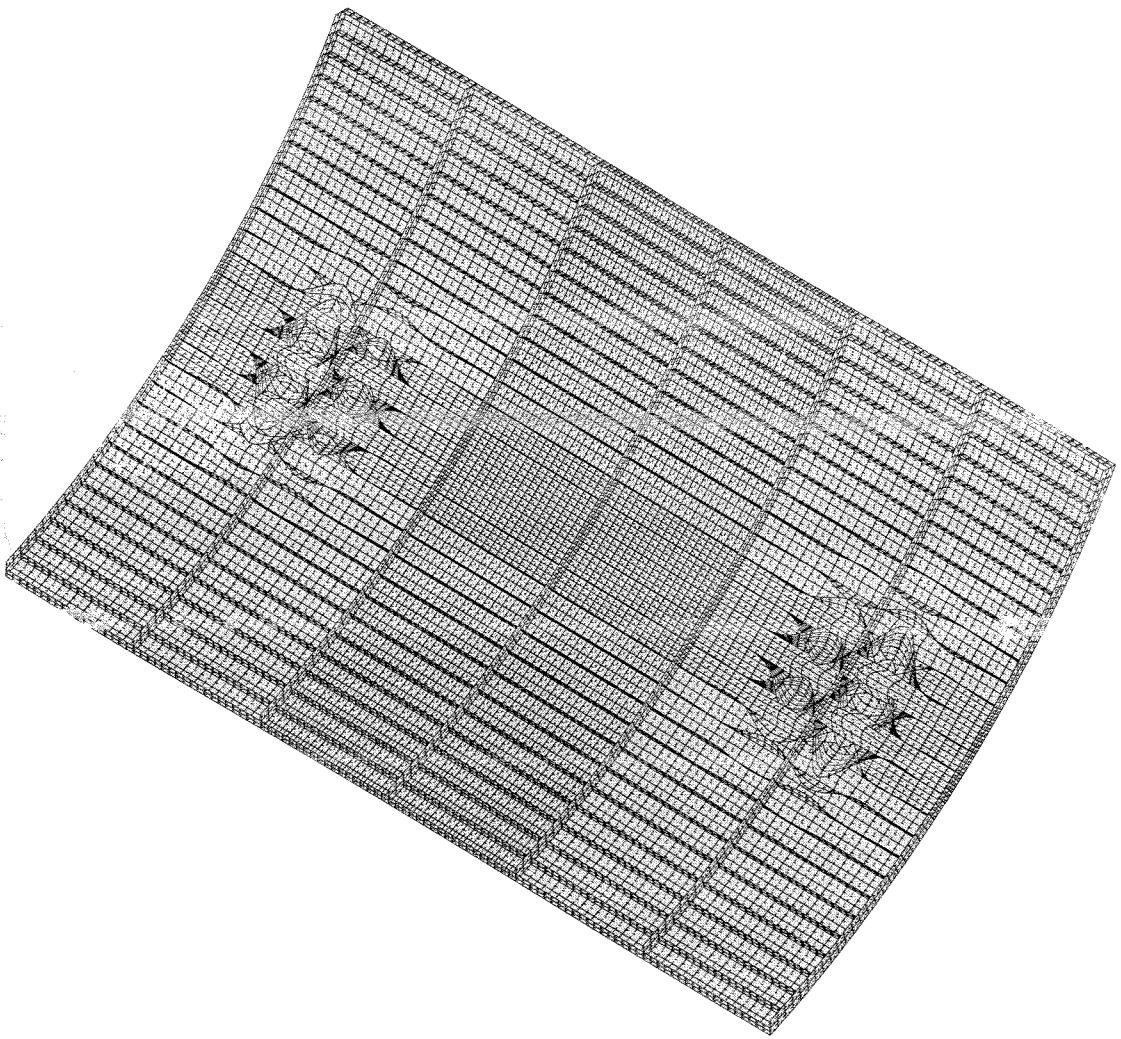
1.336E+01



183

Fig. 69

Nonlinear buckling at Load Step 9  
(PA=1.0)



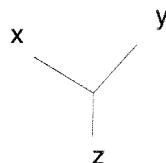
solution scale = 0.4615E+01

mode 1, pcr = 0.10430E+01

step 9 eigenvector deformed geometry  
linear buckling of perfect shell from STAGS

$\Theta_x -35.84$   
 $\Theta_y -179.86$   
 $\Theta_z 35.63$

1.336E+01



184

Fig.70

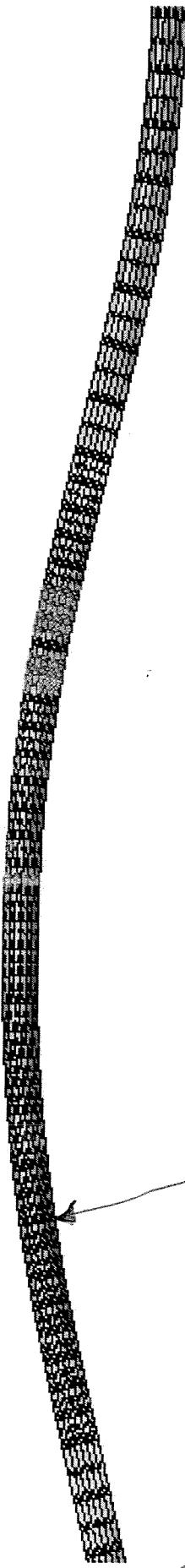


Fig. 71

02

(185)

$$PA = 1.0$$

$$).00000E+00 \quad PX = 0.00000E+00$$

$\gamma_{\text{eff}}$ , layer 1, inner fiber

inner fiber same view a linear buckling rr  
 $E+04$ , Maximum value =

3.28577E+04

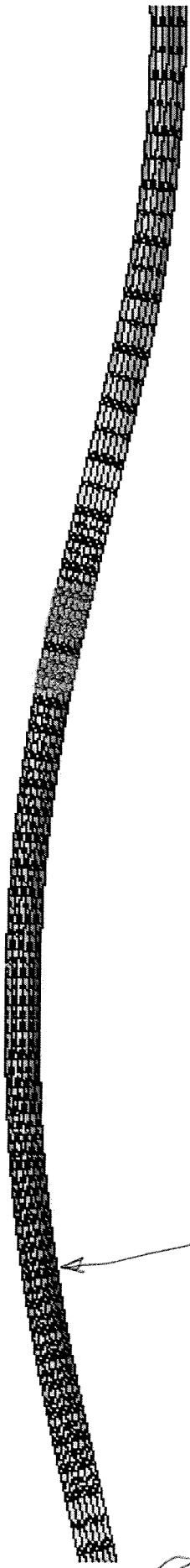


Fig. 72

(186) 2

$$PA = 1.0$$

$$00000E+00 \quad PX = 0.00000E+00$$

,seff, layer 1, outer fiber

Outer fiber same view a linear buckling mc

+04, Maximum value =  $3.31805E+04$

# Table 67 nasaantho, STG

```

n      $ Do you want a tutorial session and tutorial output?
1      $ Choose type of STAGS analysis (1,3,4,5,6), INDIC
0      $ Restart from ISTARTth load step (0=1st nonlinear soln), ISTART
1.000000 $ Local buckling load factor from PANDA2, EIGLOC
Y      $ Are the dimensions in this case in inches?
1      $ Nonlinear (0) or linear (1) kinematic relations?, ILIN
0      $ Type 1 for closed (360-deg) cyl. shell, 0 otherwise, ITOTAL
68.75000 $ X-direction length of the STAGS model of the panel: XSTAGS
50.26548 $ Panel length in the plane of the screen, L2
n      $ Is the nodal point spacing uniform along the stringer axis?
22.91667 $ Axial callout X(i) where the nodal point spacing changes, X( 1)
37     $ Number of nodes n(i) from X(i-1) to X(i) (n=odd!), n( 1)
Y      $ Are there any more interior axial stations x where dx changes?
45.83333 $ Axial callout X(i) where the nodal point spacing changes, X( 2)
73     $ Number of nodes n(i) from X(i-1) to X(i) (n=odd!), n( 2)
n      $ Are there any more interior axial stations x where dx changes?
37     $ Number of nodes n(i) from last X to x = XSTAGS, n( 3)
-2219   $ Resultant (e.g. lb/in) normal to the plane of screen, Nx
0      $ Resultant (e.g. lb/in) in the plane of the screen, Ny
0      $ In-plane shear in load set A, Nxy
0      $ Normal pressure in STAGS model in Load Set A, p
0      $ Resultant (e.g. lb/in) normal to the plane of screen, Nx0
0      $ Resultant (e.g. lb/in) in the plane of the screen, Ny0
0      $ Normal pressure in STAGS model in Load Set B, p0
1      $ Starting load factor for Load System A, STLD(1)
0      $ Load factor increment for Load System A, STEP(1)
1      $ Maximum load factor for Load System A, FACM(1)
0      $ Starting load factor for Load System B, STLD(2)
0      $ Load factor increment for Load System B, STEP(2)
0      $ Maximum load factor for Load System B, FACM(2)
8       $ How many eigenvalues do you want? NEIGS
480    $ Choose element type (410 or 411 or 480) for panel skin
n      $ Have you obtained buckling modes from STAGS for this case?
162    $ Number of stringers in STAGS model of 360-deg. cylinder
7       $ Number of rings in the STAGS model of the panel
Y      $ Are there rings at the ends of the panel?
0       $ Number of finite elements between adjacent stringers
100   $ Number of finite elements over circumference, NELCIR
9       $ Number of finite elements between adjacent rings
3       $ Stringer model: 1 or 2 or 3 or 4 or 5 (Type H(elp))
3       $ Ring model: 1 or 2 or 3 or 4 or 5 (Type H(elp))
-1     $ Reference surface of cyl: 1=outer, 0=middle, -1=inner
n      $ Do you want to use fasteners (they are like rigid links)?
n      $ Are the stringers to be "smeared out"?
n      $ Is the nodal point spacing uniform around the circumference?
24.44444 $ Circ. callout Y(i) where the nodal point spacing changes, Y( 1)
45     $ Number of nodes n(i) from Y(i-1) to Y(i) (n=odd!), n( 1)
Y      $ Are there any more interior axial stations y where dy changes?
35.55555 $ Circ. callout Y(i) where the nodal point spacing changes, Y( 2)
41     $ Number of nodes n(i) from Y(i-1) to Y(i) (n=odd!), n( 2)
n      $ Are there any more interior axial stations y where dy changes?
45     $ Number of nodes n(i) from last Y to y = YSTAGS, n( 3)
n      $ Are the rings to be "smeared out"?
5       $ Number of nodes over height of stiffener webs, NODWEB
3       $ Number of nodes over width of stringer flange, NDFLGS
3       $ Number of nodes over width of ring flange, NDFLGR
n      $ Do you want stringer(s) with a high nodal point density?
n      $ Do you want ring(s) with a high nodal point density?
n      $ Is there plasticity in this STAGS model?
Y      $ Do you want to use the "least-squares" model for torque?
Y      $ Is stiffener sidesway permitted at the panel edges?
Y      $ Do you want symmetry conditions along the straight edges?

```

input for STAGSUNIT

# Table 68 (2 pages)

nasaortholin1.out2.nonuniform2.all

OUTPUT FROM STAGS FOR LINEAR BUCKLING RUNS  
 IN SEARCH OF THE GENERAL BUCKLING MODE AND  
 LOAD FACTOR: ILIN = 1. STAGS 60-DEGREE x 6-RING-BAY MODEL  
 WITH NONUNIFORM MESH and 5 nodal points across width of  
 ALL stringers.

=====

1 nasaortho.out2 (abridged, model with nonuniform mesh)  
 shift = 2.601934, 734 negative roots

CONVERGENCE HAS BEEN OBTAINED FOR EIGENVALUES 1 THROUGH 8  
 CRITICAL LOAD FACTOR COMBINATION

NO.	EIGENVALUE	LOAD SYSTEM A	LOAD SYSTEM B	@DOF	root
1	2.598371E+00	2.598371E+00	0.000000E+00	139809	733
2	2.600594E+00	2.600594E+00	0.000000E+00	145761	734
3	2.602351E+00	2.602351E+00	0.000000E+00	154641	735
4	2.602422E+00	2.602422E+00	0.000000E+00	186537	736
5	2.603468E+00	2.603468E+00	0.000000E+00	172761	737
6	2.603567E+00	2.603567E+00	0.000000E+00	42135	738
7	2.604802E+00	2.604802E+00	0.000000E+00	136737	739
8	2.606142E+00	2.606142E+00	0.000000E+00	161481	740

2 nasaortho.out2 (abridged, model with nonuniform mesh)  
 shift = 2.595, 730 roots skipped

CONVERGENCE HAS BEEN OBTAINED FOR EIGENVALUES 1 THROUGH 8  
 CRITICAL LOAD FACTOR COMBINATION

NO.	EIGENVALUE	LOAD SYSTEM A	LOAD SYSTEM B	@DOF	root
1	2.591336E+00	2.591336E+00	0.000000E+00	186633	726
2	2.592291E+00	2.592291E+00	0.000000E+00	70143	727
3	2.593686E+00	2.593686E+00	0.000000E+00	149217	728
4	2.593722E+00	2.593722E+00	0.000000E+00	171993	729
5	2.594828E+00	2.594828E+00	0.000000E+00	149889	730
6	2.595248E+00	2.595248E+00	0.000000E+00	72963	731
7	2.595544E+00	2.595544E+00	0.000000E+00	180177	732
8	2.598371E+00	2.598371E+00	0.000000E+00	182073	733

3 nasaortho.out2 (abridged, model with nonuniform mesh)

22	1	0.25873409E+01	0.10047905E+00	0.25873752E+01
22	2	0.25882266E+01	0.72410242E-12	0.25882266E+01
22	3	0.25856329E+01	0.45478781E-11	0.25856329E+01
22	4	0.25832315E+01	0.10016660E-10	0.25832315E+01
22	5	0.25828431E+01	0.47631545E-08	0.25828431E+01
22	6	0.25913361E+01	0.46453469E-08	0.25913361E+01
22	7	0.25922910E+01	0.11332715E-06	0.25922910E+01
22	8	0.25812378E+01	0.66700105E-06	0.25812378E+01
22	9	0.25936994E+01	0.37726863E-03	0.25936994E+01
22	10	0.25937235E+01	0.20136934E-03	0.25937222E+01

shift - 2.587, 724 negative roots

MAXIMUM NUMBER OF ITERATIONS

CONVERGENCE CRITERION HAS NOT BEEN SATISFIED FOR EIGENVALUES 1 THROUGH 8  
 CRITICAL LOAD FACTOR COMBINATION

NO.	EIGENVALUE	LOAD SYSTEM A	LOAD SYSTEM B	@DOF	
1	2.581238E+00	2.581238E+00	0.000000E+00	159489	721
2	2.582843E+00	2.582843E+00	0.000000E+00	203385	722
3	2.583232E+00	2.583232E+00	0.000000E+00	116313	723
4	2.585633E+00	2.585633E+00	0.000000E+00	119241	724
5	2.587341E+00	2.587341E+00	0.000000E+00	66627	<--not converged
6	2.588227E+00	2.588227E+00	0.000000E+00	203337	725
7	2.591336E+00	2.591336E+00	0.000000E+00	135249	726
8	2.592291E+00	2.592291E+00	0.000000E+00	42279	727

4 nasaortho.out2 (abridged, model with nonuniform mesh)

22	1	0.26102443E+01	0.00000000E+00	0.26102443E+01	
22	2	0.26120537E+01	0.59001392E-11	0.26120537E+01	
22	3	0.26120585E+01	0.12585922E-11	0.26120585E+01	
22	4	0.26124634E+01	0.54621899E-10	0.26124634E+01	
22	5	0.26139468E+01	0.15381915E-08	0.26139468E+01	
22	6	0.26139518E+01	0.26770384E-07	0.26139518E+01	
22	7	0.26079914E+01	0.12103779E-10	0.26079914E+01	
root 740	22	8	0.26061421E+01	0.37639509E-03	0.26061402E+01
root 748	22	9	0.26160020E+01	0.34009980E-04	0.26160022E+01
root 749	22	10	0.26165517E+01	0.51472791E-03	0.26165545E+01
root 750	22	11	0.26167729E+01	0.26095533E-03	0.26167744E+01

shift = 2.611, 742 negative roots

MAXIMUM NUMBER OF ITERATIONS

Table 68 (p. 2 of 2)

CONVERGENCE HAS BEEN OBTAINED FOR EIGENVALUES 1 THROUGH 7  
 CONVERGENCE CRITERION HAS NOT BEEN SATISFIED FOR EIGENVALUES 8 THROUGH 8

CRITICAL LOAD FACTOR COMBINATION

NO.	EIGENVALUE	LOAD SYSTEM A	LOAD SYSTEM B	@DOF	root	
1	2.606142E+00	2.606142E+00	0.000000E+00	161481	740	<--not converged
2	2.607991E+00	2.607991E+00	0.000000E+00	215089	741	<--general buckling
3	2.610244E+00	2.610244E+00	0.000000E+00	126465	742	
4	2.612054E+00	2.612054E+00	0.000000E+00	148329	743	
5	2.612059E+00	2.612059E+00	0.000000E+00	171537	744	
6	2.612463E+00	2.612463E+00	0.000000E+00	161721	745	
7	2.613947E+00	2.613947E+00	0.000000E+00	193449	746	
8	2.613952E+00	2.613952E+00	0.000000E+00	151617	747	

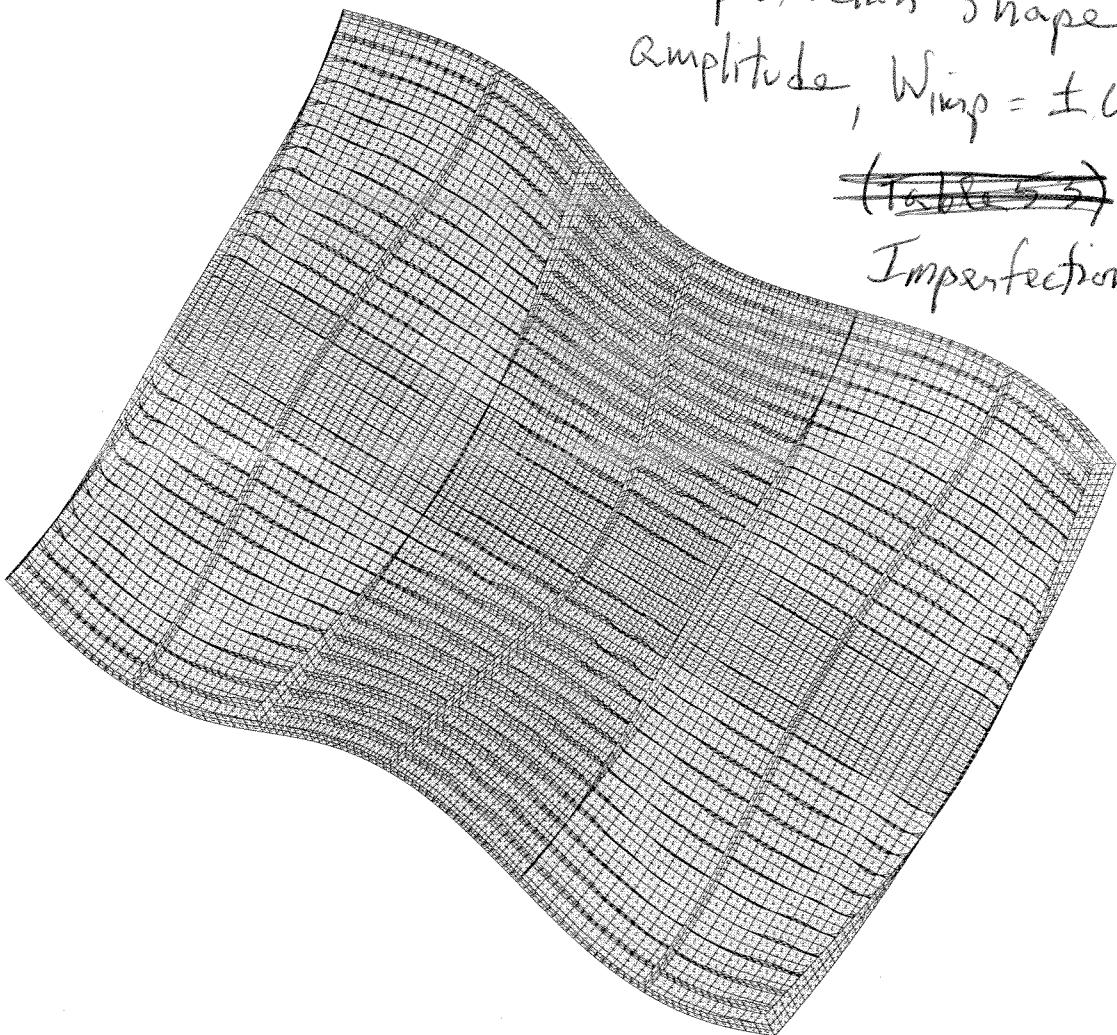
STAGS General Buckling with  $ILIN=1$   
 (60-degree model)

Compare with Fig. ~~28~~ 66

Use as imperfection shape with  
 amplitude,  $W_{imp} = \pm 0.125''$

~~(Table 53)~~

Imperfection Shape #1



solution scale = 0.4206E+01

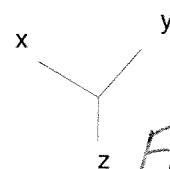
mode 1,  $\rho_{cr} = 0.26080E+01$

step 0 eigenvector deformed geometry  
 linear buckling of perfect shell from STAGS

compare with Fig. ~~28~~ 66

$\Theta_x -35.84$   
 $\Theta_y -179.86$   
 $\Theta_z 35.63$

1.336E+01



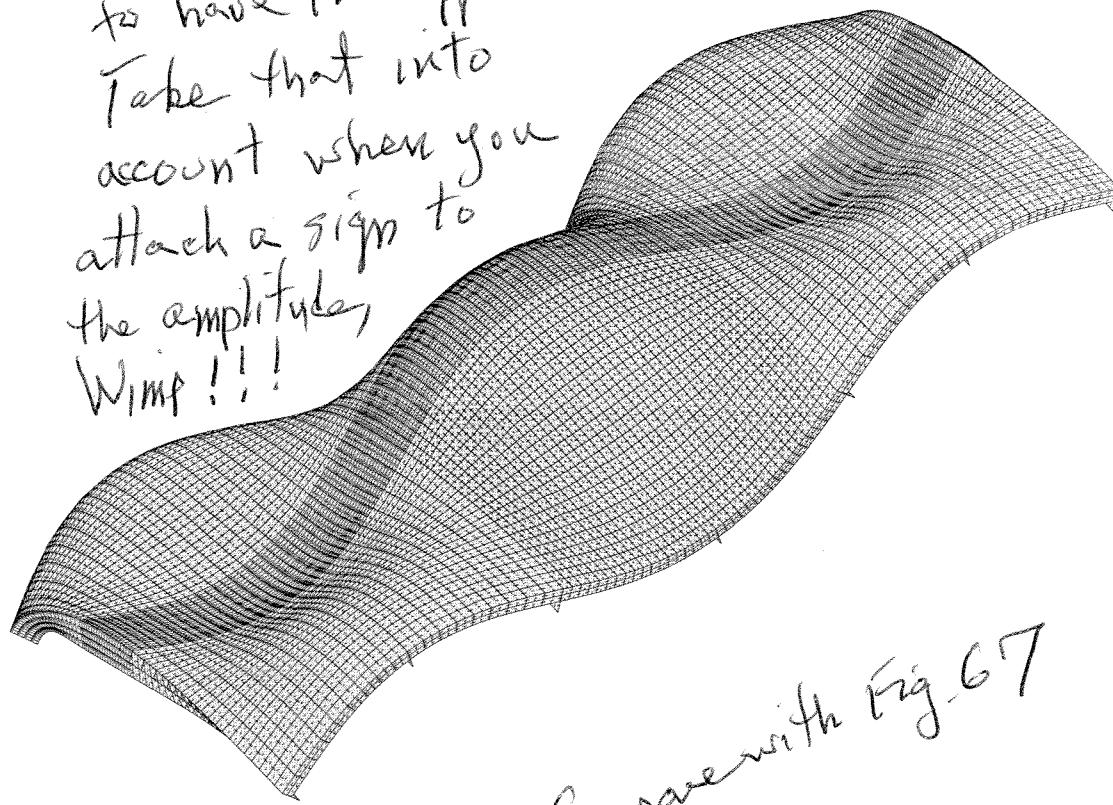
190

Fig. 73

Compare with Fig. 67.

Notice that the buckling mode happens to have the opposite sign from that in Fig. 67.

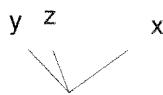
Take that into account when you attach a sign to the amplitudes  
Wimp!!!



Compare with Fig. 67

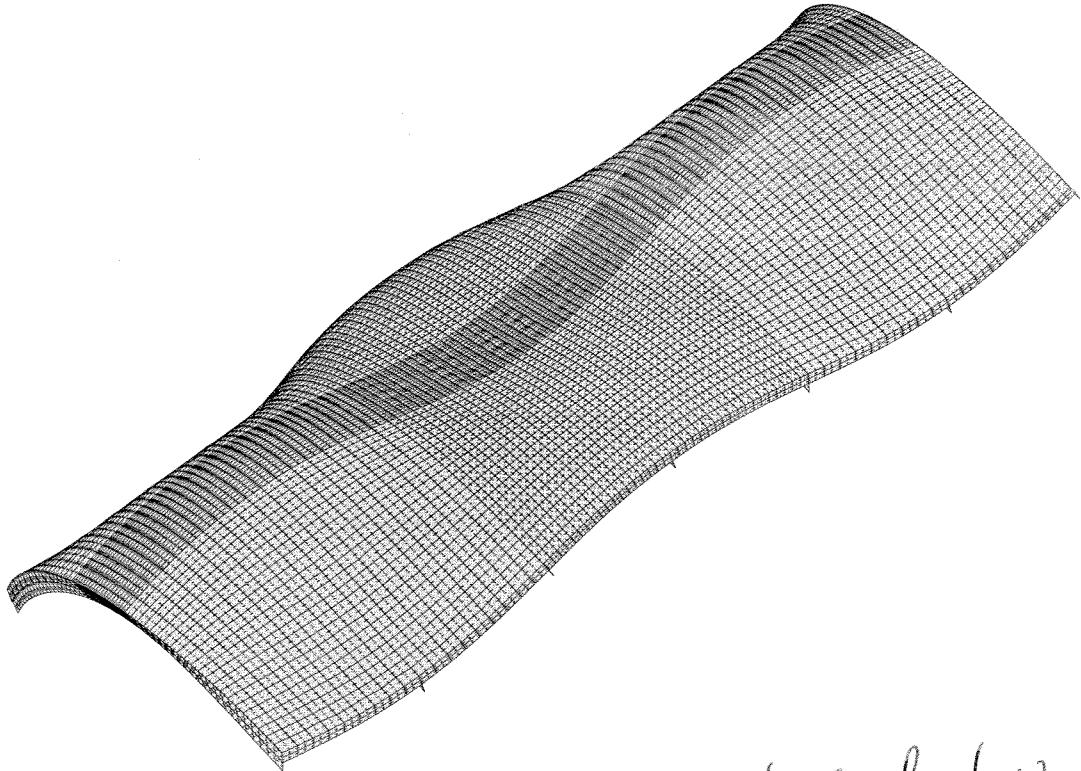
solution scale = 0.4513E+01  
mode 1, pcr = 0.26080E+01  
step 0 eigenvector deformed geometry  
linear buckling of perfect shell from STAGS

$\Theta_x -35.84$   
 $\Theta_y -13.14$   
 $\Theta_z 35.63$



1.216E+01

$$N_{imp} = -0.125''$$



$PA = 1.0$  : the design load,  $N_x = -2219 \text{ lb/in}$

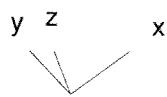
solution scale =  $0.2327E+02$

$PA = 1.00000E+00 / PB = 0.00000E+00 / PX = 0.00000E+00$

step 12 displacement deformed geometry

STAGS model: nonlinear deformation, same view as linear buckling modes

$\Theta_x -35.84$   
 $\Theta_y -13.14$   
 $\Theta_z 35.63$



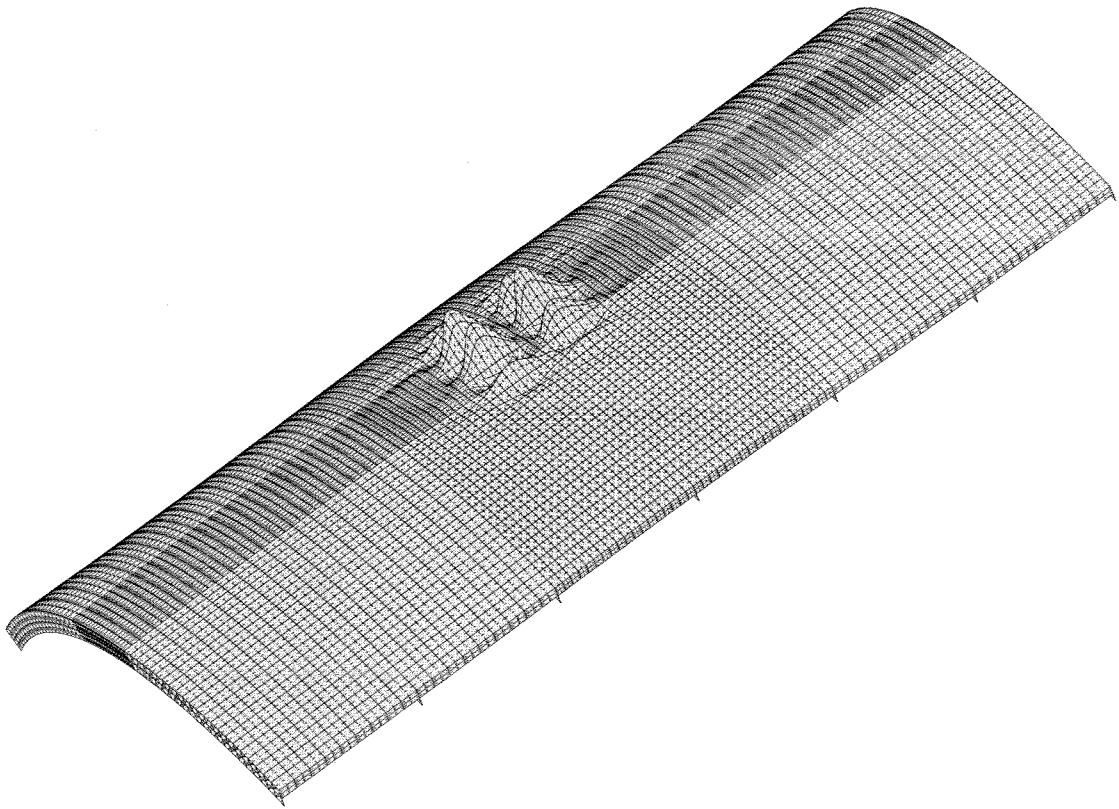
$+ 1.216E+01 +$

192

Fig. 75

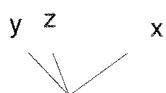
$$W_{inip} = -0.125"$$

Nonlinear local buckling  
at the design load, PA=1.0



solution scale = 0.4003E+01  
mode 1, pcr = 0.10402E+01  
step 12 eigenvector deformed geometry  
nonlinear local buckling of imperfect shell from STAGS

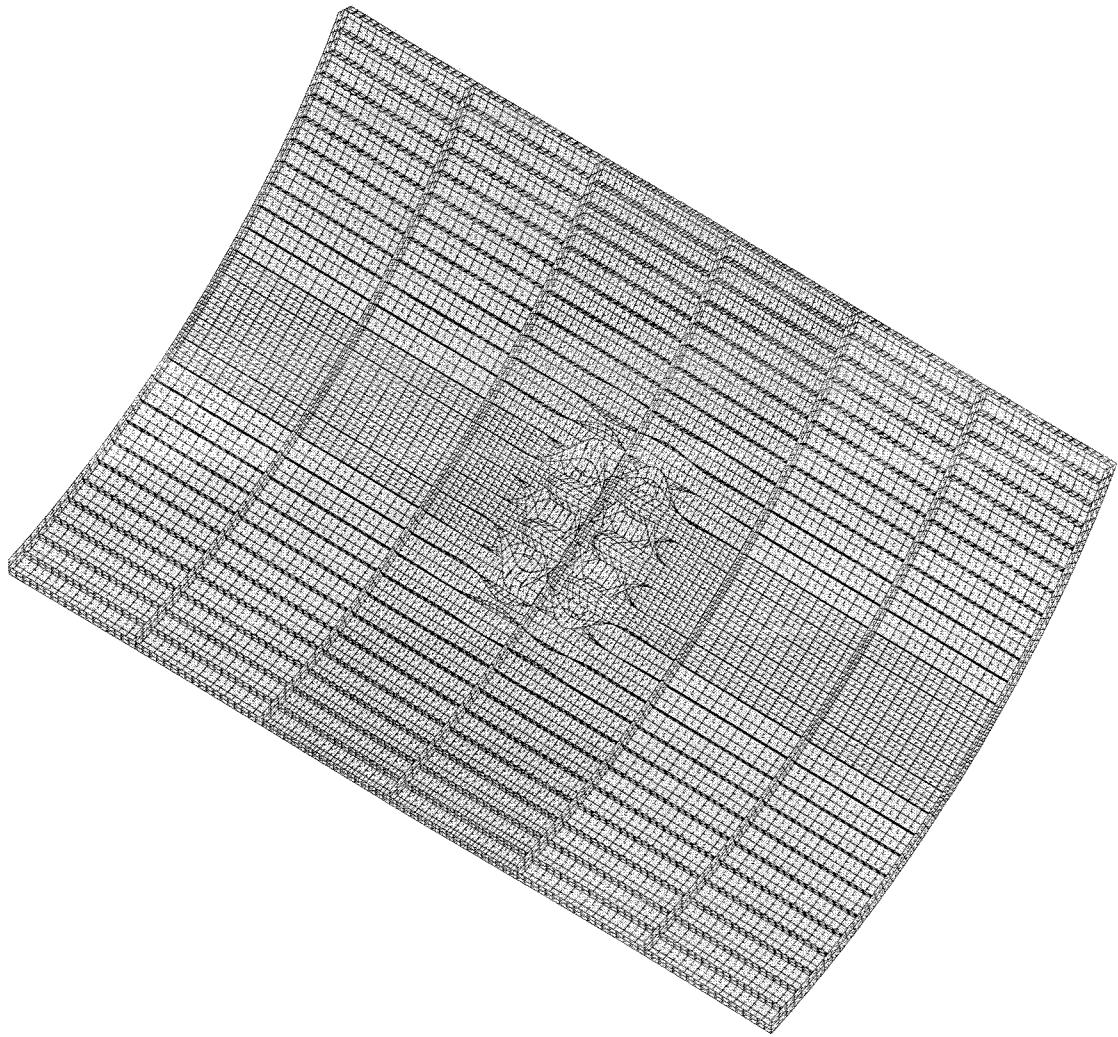
$\Theta_x$  -35.84  
 $\Theta_y$  -13.14  
 $\Theta_z$  35.63



193

Fig. 76

Same buckling mode as in the  
previous figure



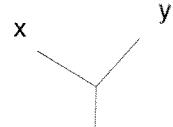
solution scale = 0.4615E+01

mode 1, pcr = 0.10402E+01

step 12 eigenvector deformed geometry

nonlinear local buckling of imperfect shell from STAGS

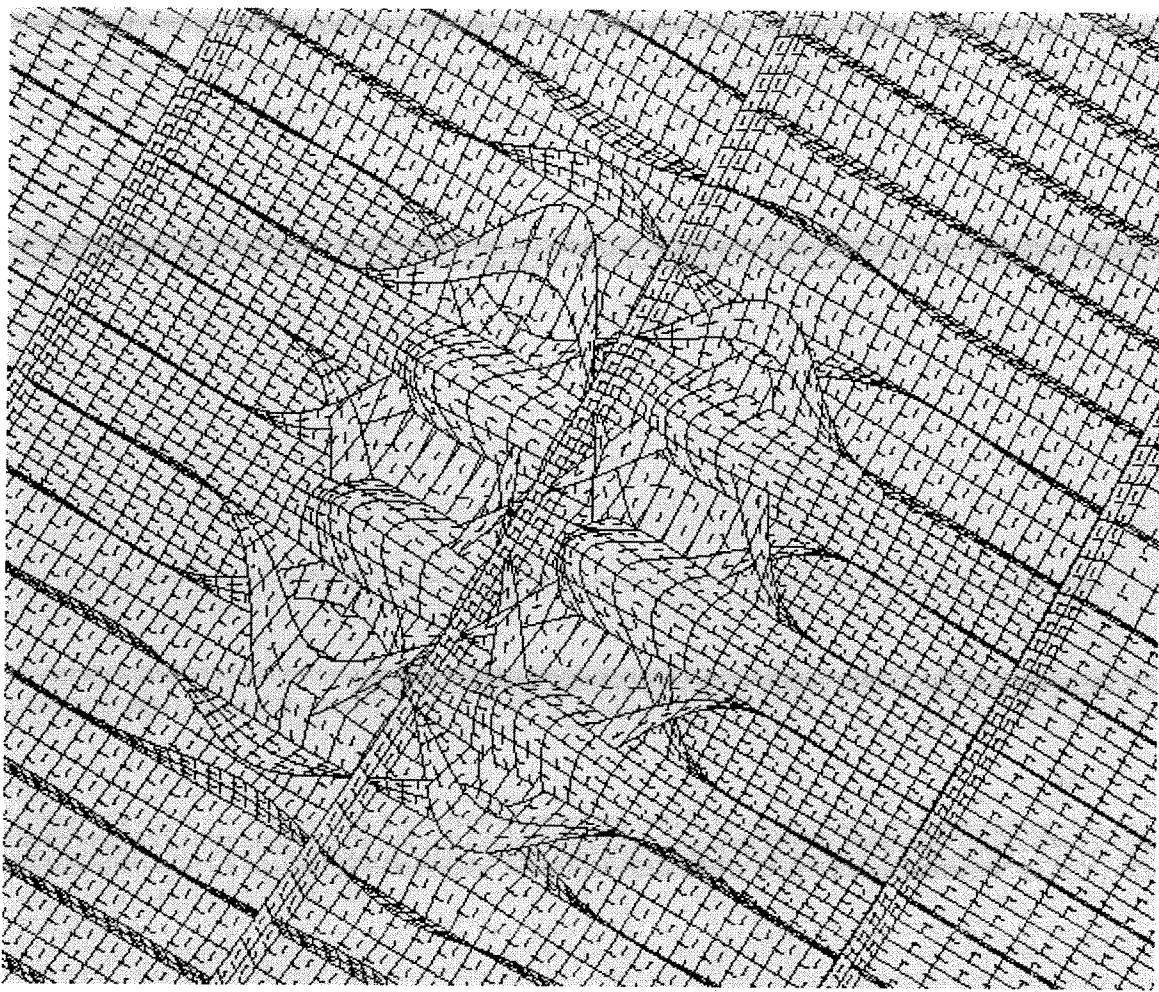
$\Theta_x -35.84$   
 $\Theta_y -179.86$   
 $\Theta_z 35.63$



194

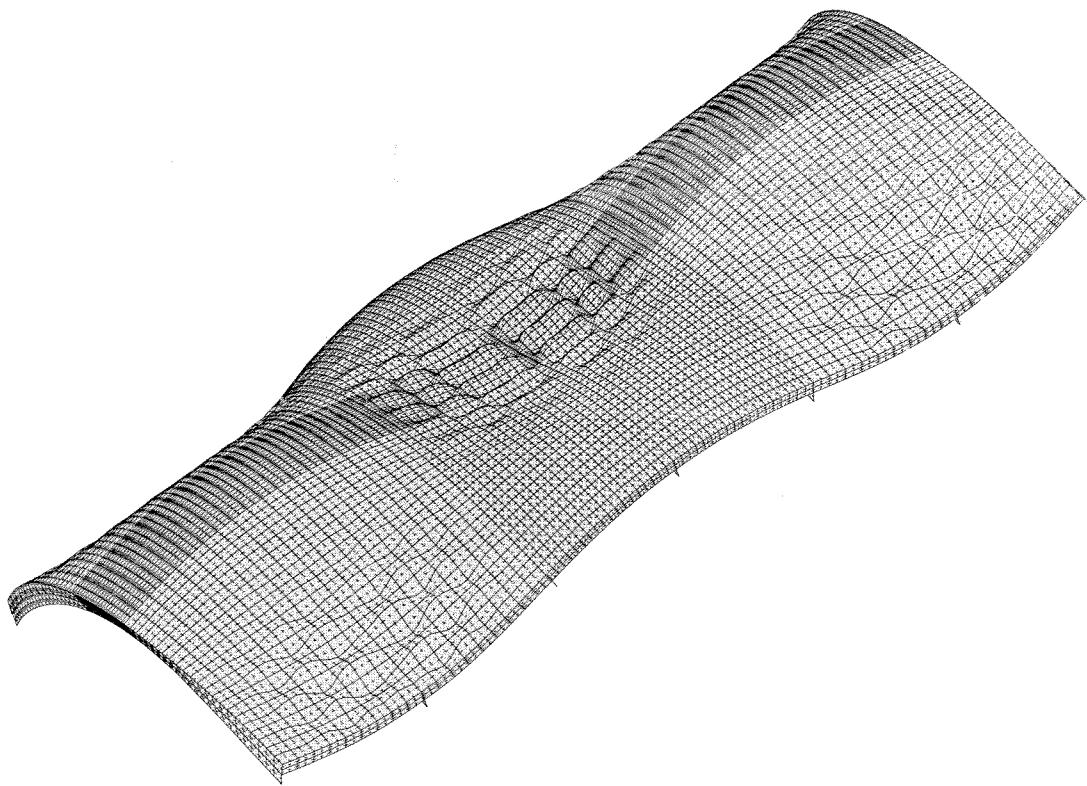
+ 1.336E+01 +

Fig. 77



Same buckling mode as in the  
previous figure

Nonlinear Equilibrium at the highest load factor, PA, is Fig. 87



solution scale = 0.1424E+02

PA= 1.27518E+00 PB= 0.00000E+00 PX= 0.00000E+00

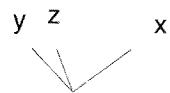
step 30 displacement deformed geometry

STAGS model: nonlinear deformation, same view as linear buckling modes

$\Theta_x -35.84$

$\Theta_y -13.14$

$\Theta_z 35.63$

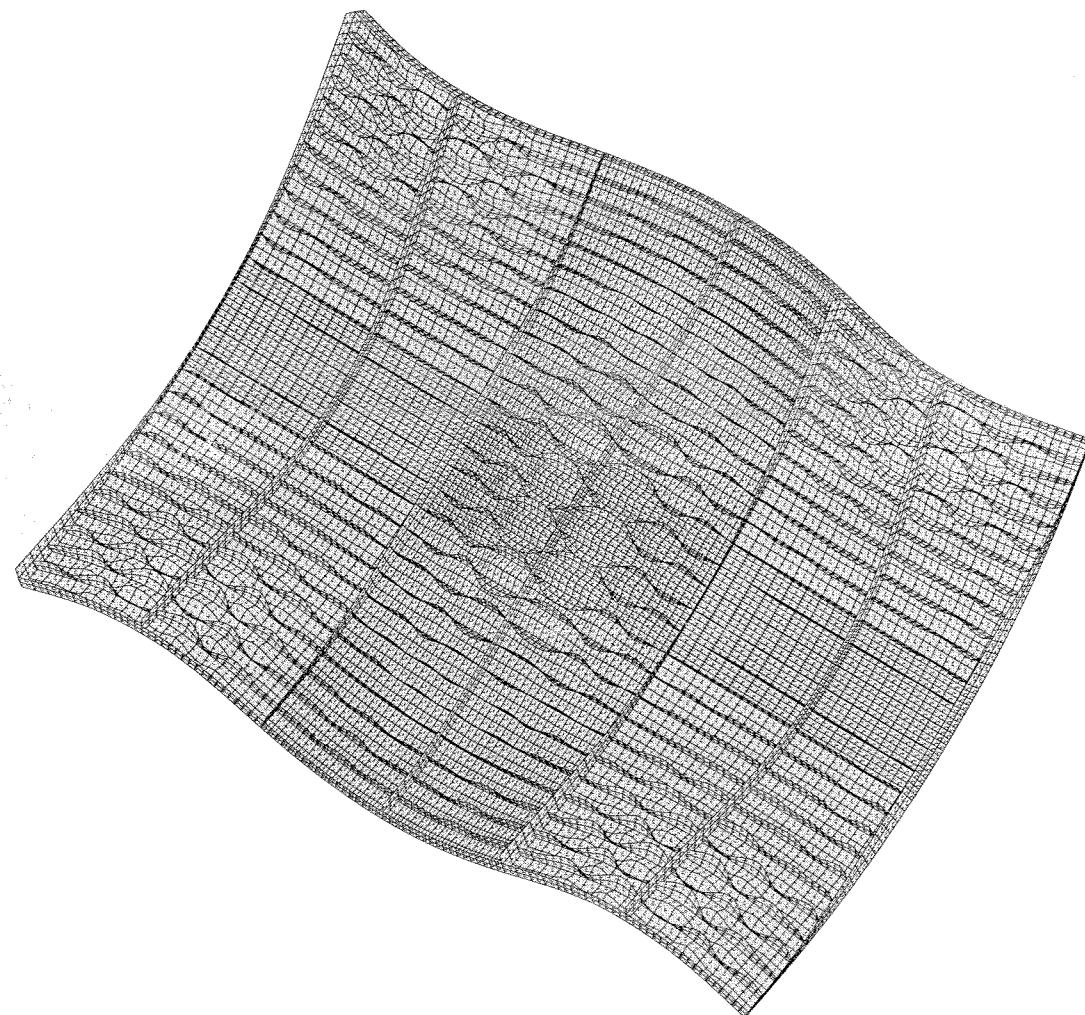


+ 1.216E+01 +

196

Fig. 79

Same nonlinear state as is shown  
in the previous Figure.



solution scale = 0.1652E+02

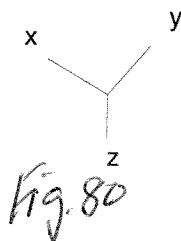
PA= 1.27518E+00 PB= 0.00000E+00 PX= 0.00000E+00

step 30 displacement deformed geometry

STAGS model: nonlinear deformation, same view as linear buckling modes

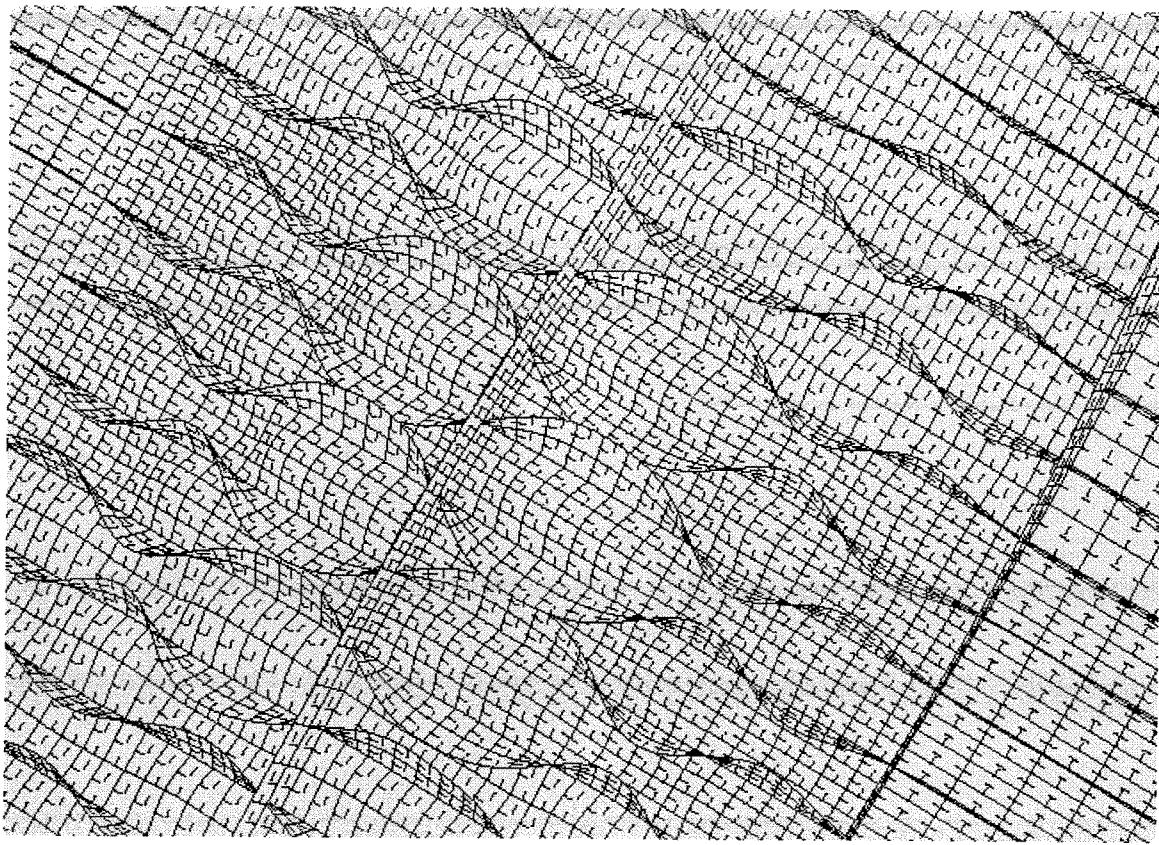
$\Theta_x -35.84$   
 $\Theta_y -179.86$   
 $\Theta_z 35.63$

1.336E+01

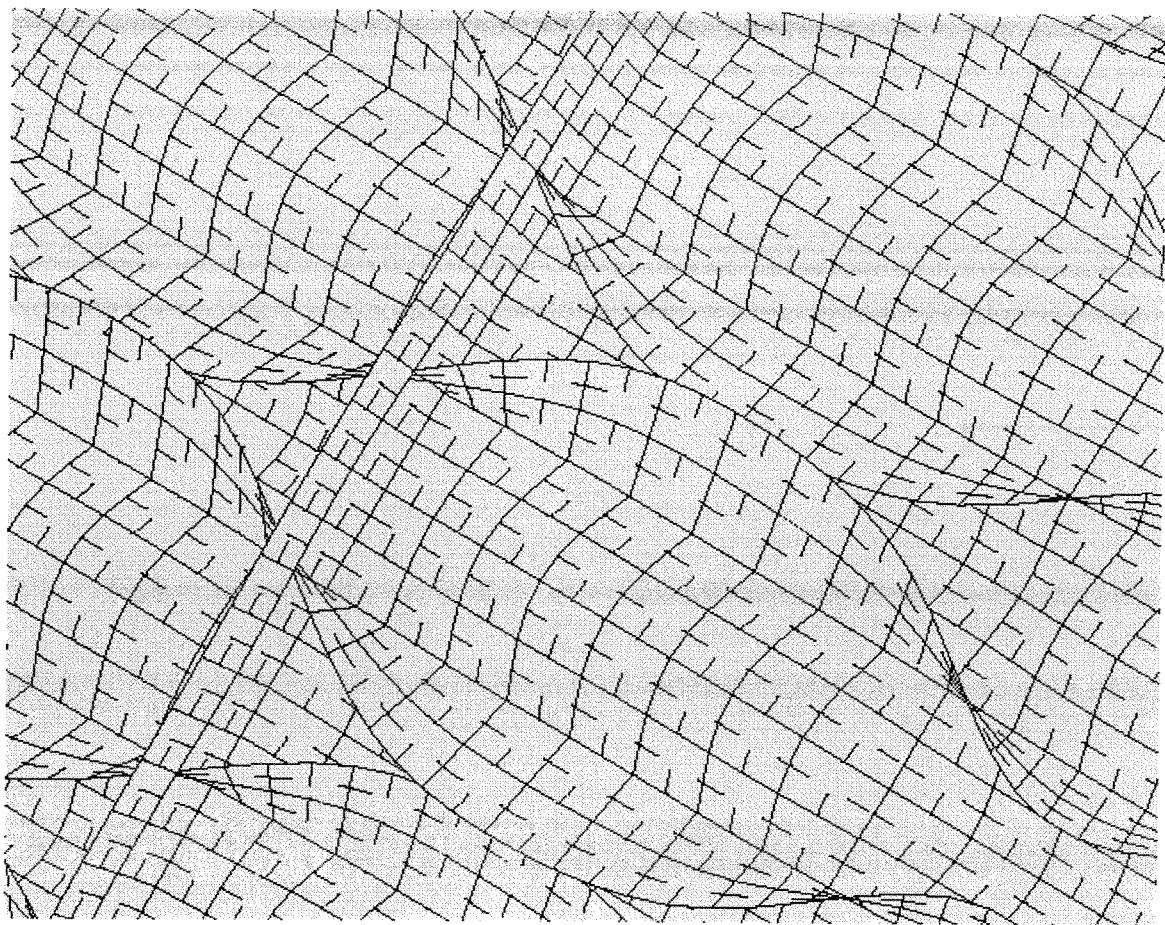


197

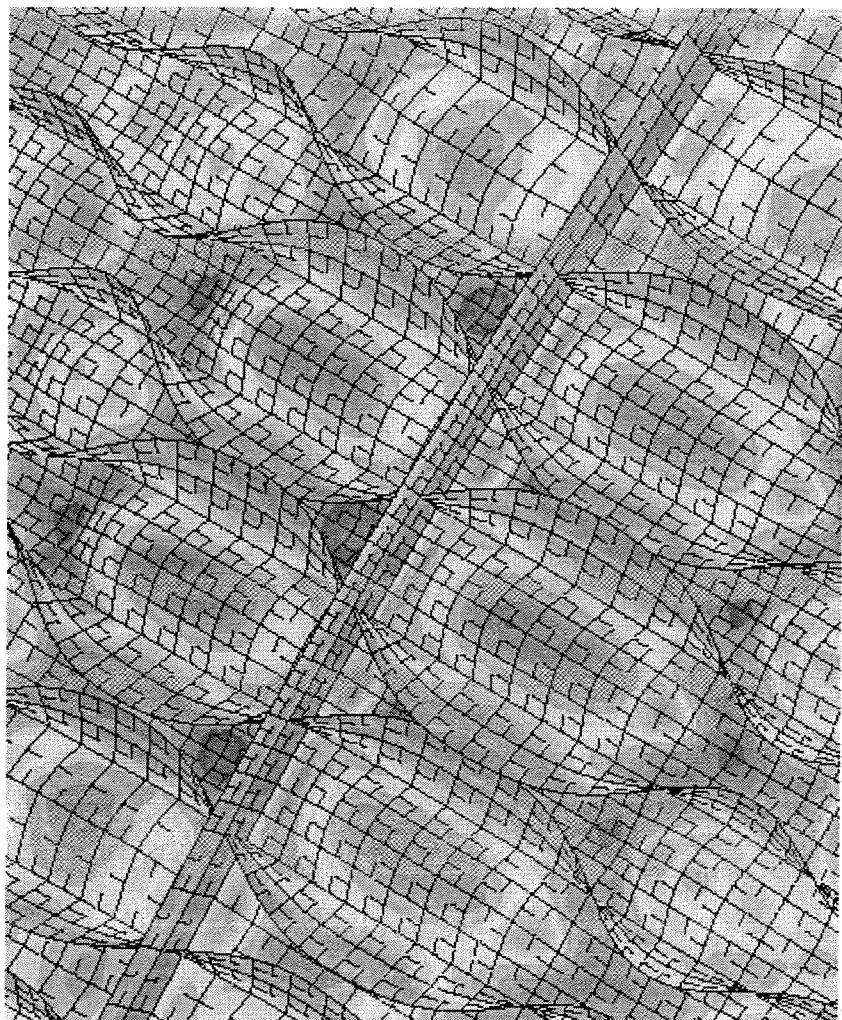
Fig. 80



Same nonlinear equilibrium state as  
is shown in the previous figure



Same nonlinear equilibrium state as is  
shown in the previous figure



inner fiber effective stress  
at  $PA = 1,275$

200

Fig. 83