



Fig. 242 The **optimized unstiffened equivalent ellipsoidal shell with the thick apex with $t(\text{apex}) = 0.61996$ inch; Wimp=0.2 inch; the optimum design is listed in Table 93. Case 1:** State of the shell at load set B (PB) step no. 42 in Run 2. (See Fig. 240). Load set B consists of a number of concentrated inward directed normal loads applied along row 2 of shell segment 3 (Figs. 2, 169, 232 and 233) distributed in the circumferential direction as $\cos(\theta)$ from $\theta = 0$ to 90 degrees. This “ $\cos(\theta)$ ” load distribution is used because it generates a dent that **locally** resembles the negative of the deformation in Fig. 232, that is, the negative of the linear buckling modal imperfection with $n = 1$ circumferential wave. Compare with Fig. 181, which pertains to the design listed in Table 78.