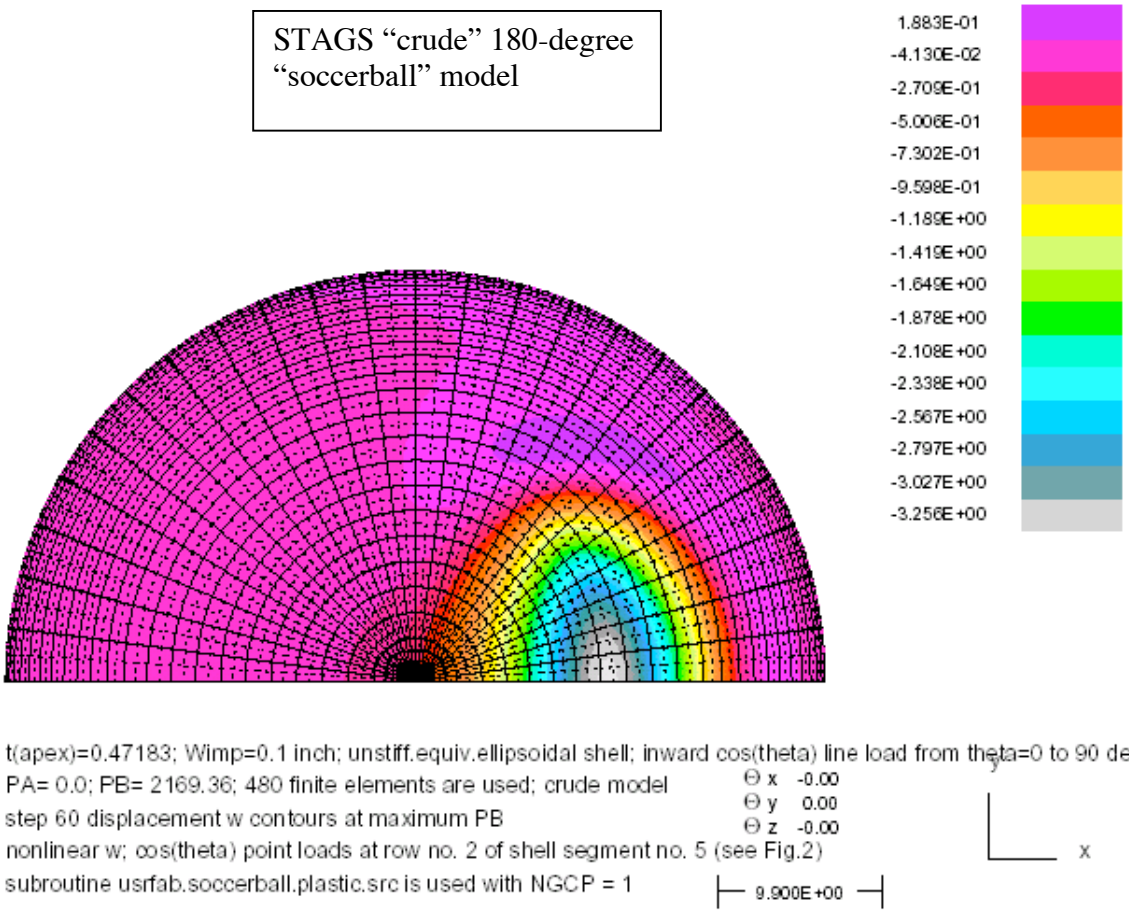

Fig. 218 Elastic-plastic analysis of the **optimized unstiffened equivalent ellipsoidal shell with the thick apex with t(apex) = 0.47183 inch; Wimp=0.1 inch**, half the amplitude, Wimp = 0.2 inch, that pertains to the results in Figs. 145 – 200 and Tables 78 – 88**; the optimum design is listed in Table 89.** State of the shell at load set B (PB) step no. 60 of Run 1. (See Fig. 216). Load set B consists of a number of concentrated inward-directed normal **loads** applied along Row 2 of Shell Segment 5 (Figs. 2, 169, 205) distributed as cos(theta) from theta = 0 to 90 degrees in the circumferential coordinate. This load distribution is used because it generates a dent that **locally** resembles the negative of the deformation in Fig. 205, that is, the linear buckling modal imperfection with n = 1 circumferential wave. Compare with Fig. 185, which applies to a different optimum design: that listed in Table 78, for which Wimp=0.2 inch.