

Table A15 List of the file, **bosdec.equivellipse**.

This is a GENOPT-user-written file that must exist if the GENOPT user's generic class of cases to be optimized makes use of the BIGBOSOR4 software. **See Table a29 for a list of the file, "howto.bosdec", which gives guidelines on how to write a valid bosdec.src file. SUBROUTINE BOSDEC produces a valid input file for BIGBOSOR4 (or for BOSOR4). This particular version of SUBROUTINE BOSDEC produces a valid BIGBOSOR4 input file called "equivellipse.ALL" corresponding to the GENOPT user's generic case called "equivellipse".**

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C=DECK      BOSDEC
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C
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C  PURPOSE IS TO SET UP BOSOR4 INPUT FILE FOR "equivellipse"
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C
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C This program was used in some (uncompleted) research I did in
C 2005 to automate the optimization of ellipsoidal tank heads
C with thickness that varies along the meridian. An ellipsoidal head
C is modelled as a number of shell segments each of which has a
C constant meridional radius of curvature. This is done in order to
C avoid element "locking" that can occur in BOSOR4 shell segments
C which have a meridional curvature that varies within a given
C shell segment.
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C
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C This technology was used to generate a BIGBOSOR4 input file for
C the ellipsoidal head under uniform internal pressure, studied
C in November, 2006.
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C
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      SUBROUTINE BOSDEC(INDX,ILOADX,INDIC,IMPERF,IFIL14,IFILE,
1              npoint,ainput,binput,LENCYL,nodes,WIMP,
1              WMODEX,xinput,xlimit,EMATL,NUMATL,DNMATL,
1              THKSKN,HIGHST,SPACNG,THSTIF,THKCYL,
1              PRESS,PMAX,N0BX,NMINBX,NMAXBX,INCRBX)
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C2345678901234567890123456789012345678901234567890123456789012
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C
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C Meaning of INDX:
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C  INDX = 1 means linear buckling of perfect shell (INDIC=1).
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C      Purpose is to obtain the axisymmetric buckling modal
C      imperfection shape, which is present in all other analyses.
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C
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C  INDX = 2 means axisymmetric collapse of imperfect shell (INDIC=0).
C                                     (Behavior no. 1: BEHX1)
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C  INDX = 3 means non-axisymmetric nonlinear bifurcation buckling
C      of imperfect shell (INDIC=1).      (Behavior no. 2: BEHX2)
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C  INDX = 4 means axisymmetric stress analysis at design load (INDIC=0).
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C      This branch yields the following behaviors:
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C      a. local buckling load factor of shell skin      (BUCKSKN). (BEHX3)
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C      b. local buckling load factor of stiffener      (BUCSTF). (BEHX4)
C      c. maximum effective stress in the shell skin (STRMAX). (BEHX5)
C      d. maximum effective stress in stiffener        (STRSTF). (BEHX6)
C      e. normal displacement at shell apex            (ENDUV).  (BEHX7)
C
C definitions of other variables in the argument list...
C      ILOADX = load case number
C      INDIC  = bigbosor4 analysis type (0 or 1 used here)
C      IMPERF = 0 no imperfection; 1 yes imperfection
C      IFIL14 = file where bigbosor4 input "deck" is stored
C      IFILE  = file where list output is accumulated
C      npoint = number of x-coordinates (including x=0 and x at equator)
C              where a segment end is provided by the user: xinput
C      ainput = semi-major axis of ellipse (ainput = xinput(npoint))
C      binput = semi-minor axis of ellipse,  $x^2/a^2 + y^2/b^2 = 1.0$ 
C      LENCYL = length of the cylindrical segment, if any
C      nodes  = number of nodal points in each segment
C      WIMP    = amplitude of initial buckling modal imperfection shape,
C              WMODEX
C      WMODEX = axisymmetric buckling modal imperfection shape
C              (obtained from bigbosor4)
C      xinput = x-coordinates corresponding to segment ends
C      xlimit = for  $x < xlimit$  use x-coordinate for callouts
C              for  $x > xlimit$  use y-coordinate for callouts
C      EMATL  = elastic modulus of isotropic material
C      NUMATL = Poisson ratio of isotropic material
C      DNMATL = mass density of isotropic material
C      THKSKN = skin thickness corresponding to xinput
C      HIGHST = stiffener height corresponding to xinput
C      SPACNG = isogrid spacing
C      THSTIF = isogrid member thickness
C      THKCYL = thickness of cylindrical segment, if any
C      PRESS(ILOADX) = applied pressure for load case ILOADX
C      PMAX = maximum pressure to be applied
C      NOBX  = starting circ. wavenumber for buckling analysis
C      NMINBX = minimum circ. wavenumber for buckling analysis
C      NMAXBX = maximum circ. wavenumber for buckling analysis
C      INCRBX = increment in circ. wavenumber for buckling analysis
C
C      COMMON/NUMPR2/ILAR,ICAR,IOAR,NFLAT,NCASES,NPRINT
C      real LENCYL,NUMATL
C      double precision x,y,phi,r,rknuck,a1,a2,b1,b2,x03,y03
C      double precision x1,y1,x2,y2,x3,y3,a,b,r1,r2
C      dimension x(21),y(21),x1(20),y1(20),x2(20),y2(20),x3(20),y3(20)
C      dimension r1(20),r2(20)
C      dimension THKSKN(21),HIGHST(21)
C      dimension PRESS(*),WMODEX(*),xinput(21),NMESH(20)

```

C

REWIND IFIL14

C

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      IF (NPRINT.GE.2) WRITE(IFILE,3)
3  FORMAT(//' ***** BOSDEC *****'/
1'  The purpose of BOSDEC is to set up an input file, NAME.ALL, '/
1'  for equivalent ellipsoidal shell. NAME is your name for'/
1'  the case. The file NAME.ALL is a BOSOR4 input "deck" used'/
1'  by SUBROUTINE B4READ.'/
1'  *****'/)
```

C

c This version of SUBROUTINE BOSDEC is for an "equivalent" ellipsoidal head.

c The "equivalent" ellipsoidal head is constructed because BOSOR4 (bigbosor4)

c finite elements tend to "lock up" for shells of revolution in which the meridional curvature varies significantly within a single shell segment.

c

c The "equivalent" ellipsoidal head consists of a user-defined number of toroidal segments that match as well as possible the contour of the ellipsoidal head. The meridional curvature of each toroidal segment is constant in that segment. Therefore, there is no problem of finite element "lock up" in a segmented model of this type.

c

c For each toroidal segment, bigbosor4 needs three points for input:

c (x1,y1), (x2,y2), and (x3,y3). (x1,y1) and (x2,y2) lie on the ellipsoidal

c contour and are the (x,y) coordinates at the two ends of the toroidal segment. (x3,y3) is the center of meridional curvature of the toroidal segment. The trick is to obtain (x3,y3) so that to toroidal segment best fits the ellipsoidal contour in that segment.

c

c We use the following procedure to get (x3,y3):

c

c 1. The equation of the ellipse is

c

$$x^2/a^2 + y^2/b^2 = 1.0 \quad (1)$$

c

c 2. The equation for the normal to the ellipse at (x1,y1) is:

c

$$y - y1 = (y1/x1)(a^2/b^2)(x - x1) \quad (2)$$

c

c 3. The equation for the normal to the ellipse at (x2,y2) is:

c

$$y - y2 = (y2/x2)(a^2/b^2)(x - x2) \quad (3)$$

c

c 4. These two straight lines in (x,y) space intersect at (x03,y03), with (x03,y03) are given by:

$$x03 = (b2 - b1)/(a1 - a2); \quad y03 = (a2*b1 - a1*b2)/(a2 - a1) \quad (4)$$

c in which  $a_1$ ,  $b_1$  and  $a_2$ ,  $b_2$  are:

c  
c  $a_1 = (y_1/x_1)(a^2/b^2); \quad b_1 = -a_1*x_1 + y_1 \quad (5)$

c  $a_2 = (y_2/x_2)(a^2/b^2); \quad b_2 = -a_2*x_2 + y_2 \quad (6)$

c  
c 5. For an ellipse the distance from the point  $(x_3, y_3)$  to  $(x_1, y_1)$  is  
c different than the distance from the point  $(x_3, y_3)$  to  $(x_2, y_2)$   
c because the meridional curvature varies along the contour of the  
c ellipse. We wish to find a new point  $(x_3, y_3)$  in the neighborhood  
c of  $(x_3, y_3)$  for which the distance from  $(x_3, y_3)$  to  $(x_1, y_1)$  equals  
c the distance from  $(x_3, y_3)$  to  $(x_2, y_2)$ . For such a point the  
c "equivalent" segment will be a toroidal segment in which the  
c meridional curvature is constant along the segment arc.

c  
c 6. The square of the distances from  $(x_3, y_3)$  to  $(x_1, y_1)$  and to  $(x_2, y_2)$   
c are:

c  $d1sq = (x_1 - x_3)^2 + (y_1 - y_3)^2 \quad (7)$

c  $d2sq = (x_2 - x_3)^2 + (y_2 - y_3)^2 \quad (8)$

c  
c and the difference of these is:

c  $delsq = d1sq - d2sq \quad (9)$

c  
c 7. We determine the location of the center of meridional curvature of  
c the "equivalent" toroidal segment by allocating half of  $delsq$  to  
c each  $(distance)^2$ ,  $d1sq$  and  $d2sq$ . We then have two  $(distance)^2$   
c that are equal:

c  $(x_1 - x_3)^2 + (y_1 - y_3)^2 - delsq/2 \quad (10)$

c  $(x_2 - x_3)^2 + (y_2 - y_3)^2 + delsq/2 \quad (11)$

c  
c 8. Suppose we let

c  $x_3 = x_3 + dx \quad ; \quad y_3 = y_3 + dy \quad (12)$

c  
c Then we have two nonlinear equations for the unknowns  $(dx, dy)$ :

c  $[x_1 - (x_3+dx)]^2 + [y_1 - (y_3+dy)]^2 =$   
c  $(x_1 - x_3)^2 + (y_1 - y_3)^2 - delsq/2 \quad (13)$

c  $[x_2 - (x_3+dx)]^2 + [y_2 - (y_3+dy)]^2 =$   
c  $(x_2 - x_3)^2 + (y_2 - y_3)^2 + delsq/2 \quad (14)$

c  
c These two equations say that the square of the distance from  
c  $(x_3, y_3)$  to  $(x_1, y_1)$  Eq.(13) is equal to that from  $(x_3, y_3)$  to  $(x_2, y_2)$   
c Eq.(14).  
c

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c 9. We use Newton's method to solve the two simultaneous nonlinear
c   equations for (dx,dy):
c
c   For the ith Newton iteration, let
c
c       dx(i) = dx(i-1) + u                               (15)
c       dy(i) = dy(i-1) + v                               (16)
c
c   Then we develop two linear equations for u and v for the ith
c   Newton iteration:
c
c       u*2.*(x03-x1+dx(i-1)) +v*2.*(y03-y1 +dy(i-1)) = f1pp      (17)
c       u*2.*(x03-x2+dx(i-1)) +v*2.*(y03-y2 +dy(i-1)) = f2pp      (18)
c
c   in which the right-hand sides, f1pp and f2pp, are rather long
c   expressions given in SUBROUTINE x3y3, where the Newton iterations
c   occur.
c
c   Now find (x3,y3)...
c
c   Get end points (x1,y1), (x2,y2), and center of curvature (x3,y3)
c   of each shell segment in the model...
c
c   first, given x, get y...
c   the y are obtained from the equation for an ellipse:  $x^2/a^2 + y^2/b^2 = 1$ 
c
c       a = ainput
c       b = binput
c       do 10 i = 1,npoint
c           x(i) = xinput(i)
c           y(i) = -b*dsqrt(1.-x(i)**2/a**2)
c   10 continue
c
c   the endpoints of the first segment (bottom of "ellipse") are
c
c       r = a**2/b
c       x1(1) = 0.
c       y1(1) = -b
c       x2(1) = x(2)
c       phi = dasin(x(2)/r)
c       y2(1) = r*(1 - dcos(phi)) - b
c       x3(1) = 0.
c       y3(1) = r - b
c
c   the endpoints of the last segment (nearest the equator) are
c
c       nseg = npoint - 1

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    rknucl = b**2/a
    x1(nseg) = x(npnt-1)
    phi = dacos((x(npnt-1) - a + rknucl)/rknucl)
    y1(nseg) = -rknucl*dsin(phi)
    x2(nseg) = a
    y2(nseg) = 0.
    x3(nseg) = a -rknucl
    y3(nseg) = 0.
c
c next, establish the endpoints and centers of curvature of
c shell segments 2 - (nseg-1)
c
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    if (NPRINT.GE.2) write(ifile,'(/,A,A,I3,A,/,A,A)')
1' End points (x1,y1), (x2,y2) and center of curvature, (x3,y3)',
1' for',nseg,' toroidal segments',
1' Seg.      x1          y1          x2          y2          x3',
1'          y3          r1          r2'
    iseg = 1
c
    r1(iseg) = dsqrt((x1(iseg) - x3(iseg))**2
1          +(y1(iseg) - y3(iseg))**2)
    r2(iseg) = dsqrt((x2(iseg) - x3(iseg))**2
1          +(y2(iseg) - y3(iseg))**2)
c
    if (NPRINT.GE.2) write(ifile,'(I3,1P,8E12.4)')
1 iseg,x1(iseg),y1(iseg),x2(iseg),y2(iseg),x3(iseg),y3(iseg),
1    r1(iseg),r2(iseg)
    do 1000 iseg = 2,nseg
        iseg1 = iseg - 1
        x1(iseg) = x2(iseg1)
        y1(iseg) = y2(iseg1)
        ipnt = iseg + 1
        x2(iseg) = x(ipnt)
        y2(iseg) = y(ipnt)
c find point, (x03,y03), where the normals to the ellipse at
c (x1,y1) and (x2,y2) intersect.
    a1 = y1(iseg)*a**2/(x1(iseg)*b**2)
    a2 = y2(iseg)*a**2/(x2(iseg)*b**2)
    b1 = -a1*x1(iseg) + y1(iseg)
    b2 = -a2*x2(iseg) + y2(iseg)
    x03 = (b2 - b1)/(a1 - a2)
    y03 = (a2*b1 - a1*b2)/(a2 - a1)
c
c we wish to replace the ellipse with an "equivalent" ellipse.
c the "equivalent" ellipse consists of a number of torispherical
c segments with end points (x1,y1) and (x2,y2) and center of
c curvature (x3,y3). The purpose of subroutine x3y3 is to

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c  determine (x3,y3) given (x1,y1), (x2,y2), and (x03,y03).
c
      call x3y3(ifile,iseq,x1(iseq),y1(iseq),x2(iseq),y2(iseq),
1         x03,y03, x3(iseq),y3(iseq))
c
      r1(iseq) = dsqrt((x1(iseq) - x3(iseq))**2
1         +(y1(iseq) - y3(iseq))**2)
      r2(iseq) = dsqrt((x2(iseq) - x3(iseq))**2
1         +(y2(iseq) - y3(iseq))**2)
c
      if (NPRINT.GE.2) write(ifile,'(I3,1P,8E12.4)')
1  iseq,x1(iseq),y1(iseq),x2(iseq),y2(iseq),x3(iseq),y3(iseq),
1  r1(iseq),r2(iseq)
c
1000 continue
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c
      IF (INDIC.EQ.0.AND.INDX.EQ.4) WRITE(IFIL14,'(A)')
1' Nonlinear axisymmetric stress analysis (INDIC=0)'
      IF (INDIC.EQ.0.AND.INDX.EQ.2) WRITE(IFIL14,'(A)')
1' Nonlinear axisymmetric collapse analysis (INDIC=0)'
      IF (INDIC.EQ.1) WRITE(IFIL14,'(A)')
1' Bifurcation buckling analysis (INDIC=1)'
C BEG MAR 2008
      IF (INDIC.EQ.-2) WRITE(IFIL14,'(A)')
1' Bifurcation buckling analysis (INDIC=-2)'
C END MAR 2008
      IF (INDIC.EQ.2) WRITE(IFIL14,'(A)')
1' Modal vibration of prestressed shell'
      WRITE(IFIL14,'(I3,A)') INDIC, '          $ INDIC'
      WRITE(IFIL14,'(A)') ' 1          $ NPRT'
      ISTRES = 0
      IF (INDIC.EQ.0) ISTRES = 1
      WRITE(IFIL14,'(I3,A)') ISTRES, '          $ ISTRES'
      IF (LENCYL.GT.0.001)
1 WRITE(IFIL14,'(I4,A)') nseg+1, '          $ nseg'
      IF (LENCYL.LE.0.001)
1 WRITE(IFIL14,'(I4,A)') nseg, '          $ nseg'
c
C Begin loop over Segment data
C
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      IALL = 0
      Do 2000 iseq = 1,nseg
          NMESH(iseq) = nodes
          WRITE(IFIL14,'(I4,A)') NMESH(iseq), '          $ NMESH'
          WRITE(IFIL14,'(A)') ' 3          $ NTYPEH'
          WRITE(IFIL14,'(A)') ' 2          $ NSHAPE'

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WRITE(IFIL14,'(1P,E14.6,A)') x1(iseq), ' $ R1'
WRITE(IFIL14,'(1P,E14.6,A)') y1(iseq), ' $ Z1'
WRITE(IFIL14,'(1P,E14.6,A)') x2(iseq), ' $ R2'
WRITE(IFIL14,'(1P,E14.6,A)') y2(iseq), ' $ Z2'
WRITE(IFIL14,'(1P,E14.6,A)') x3(iseq), ' $ RC'
WRITE(IFIL14,'(1P,E14.6,A)') y3(iseq), ' $ ZC'
WRITE(IFIL14,'(A)') -1. $ SROT'
WRITE(IFIL14,'(I4,A)') IMPERF, ' $ IMP'
IF (IMPERF.EQ.1) THEN
  WRITE(IFIL14,'(A)') 4 $ ITYPE'
  WRITE(IFIL14,'(1P,E14.6,A)') WIMP, ' $ WIMP'
  WRITE(IFIL14,'(A)') 1 $ ISTART'
  NUMB = NMESH(iseq) + 2
  WRITE(IFIL14,'(I4,A)') NUMB, ' $ NUMB'
  DO 5 I = 1,NUMB
    J = I + IALL
    WRITE(IFIL14,'(1P,E14.6,A)') WMODEX(J), ' $ WSHAPE'
5    CONTINUE
    WRITE(IFIL14,'(A)') N $ any more modes?'
  ENDIF
  WRITE(IFIL14,'(A)') 3 $ NTYPEZ'
  WRITE(IFIL14,'(A)') 0. $ ZVAL'
  WRITE(IFIL14,'(A)') Y $ print r(s)...?'
  WRITE(IFIL14,'(A)') 0 $ NRINGS'
  WRITE(IFIL14,'(A)') 0 $ K'
  WRITE(IFIL14,'(A)') 0 $ LINTYP'
  WRITE(IFIL14,'(A)') 1 $ IDISAB'
  WRITE(IFIL14,'(A)') 1 $ NLTYPE'
  WRITE(IFIL14,'(A)') 2 $ NPSTAT'
  WRITE(IFIL14,'(A)') 0 $ NLOAD(1)'
  WRITE(IFIL14,'(A)') 0 $ NLOAD(2)'
  WRITE(IFIL14,'(A)') 1 $ NLOAD(3)'
  WRITE(IFIL14,'(A)') -1. $ PN(1)'
  WRITE(IFIL14,'(A)') -1. $ PN(2)'
  IF (x1(iseq).le.xlimit) then
    ntype = 3
    call1 = x1(iseq)
    call2 = x2(iseq)
  else
    ntype = 2
    call1 = y1(iseq)
    call2 = y2(iseq)
  endif
  WRITE(IFIL14,'(I4,A)') ntype, ' $ ntype'
  WRITE(IFIL14,'(1P,E14.6,A)') call1, ' $ callout1'
  WRITE(IFIL14,'(1P,E14.6,A)') call2, ' $ callout2'
  WRITE(IFIL14,'(A)') 10 $ NWALL'
  WRITE(IFIL14,'(A)') 2 $ NWALL2'

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WRITE(IFIL14,'(1P,E14.6,A)') EMATL, ' $ E'
WRITE(IFIL14,'(1P,E14.6,A)') NUMATL, ' $ U'
WRITE(IFIL14,'(1P,E14.6,A)') DNMATL, ' $ SM'
WRITE(IFIL14,'(A)')' 0. $ ALPHA'
WRITE(IFIL14,'(A)')' 1 $ NRS'
WRITE(IFIL14,'(A)')' -1 $ NSUR'
WRITE(IFIL14,'(A)')' 1 $ NTYPET'
IRADTH = 2
WRITE(IFIL14,'(I4,A)') IRADTH, ' $ NTVALU'
WRITE(IFIL14,'(I4,A)') ntype, ' $ ntype'
WRITE(IFIL14,'(1P,E14.6,A)') call1, ' $ callout1'
WRITE(IFIL14,'(1P,E14.6,A)') call2, ' $ callout2'
ipoint = iseg + 1
WRITE(IFIL14,'(1P,E14.6,A)') THKSKN(iseg), ' $ THKSKN(iseg)'
WRITE(IFIL14,'(1P,E14.6,A)') THKSKN(ipoint), ' $ THKSKN(ipoint)'
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WRITE(IFIL14,'(A)')' Y $ print refsurf...?'
WRITE(IFIL14,'(A)')' Y $ are there stringers or isogrid...?'
WRITE(IFIL14,'(A)')' 0 $ K1 (0 means internal)'
WRITE(IFIL14,'(1P,E14.6,A)') EMATL, ' $ E'
WRITE(IFIL14,'(1P,E14.6,A)') NUMATL, ' $ U'
WRITE(IFIL14,'(1P,E14.6,A)') DNMATL, ' $ SM'
WRITE(IFIL14,'(1P,E14.6,A)') SPACNG, ' $ isogrid spacing'
WRITE(IFIL14,'(A)')' N $ constant cross section?'
WRITE(IFIL14,'(I4,A)') IRADTH, ' $ number of callouts'
WRITE(IFIL14,'(I4,A)') ntype, ' $ ntype'
WRITE(IFIL14,'(1P,E14.6,A)') call1, ' $ callout1'
WRITE(IFIL14,'(1P,E14.6,A)') call2, ' $ callout2'
WRITE(IFIL14,'(1P,E14.6,A)') THSTIF, ' $ THSTIF'
WRITE(IFIL14,'(1P,E14.6,A)') THSTIF, ' $ THSTIF'
WRITE(IFIL14,'(1P,E14.6,A)') HIGHST(iseg), ' $ HIGHST(iseg)'
WRITE(IFIL14,'(1P,E14.6,A)') HIGHST(ipoint), ' $ HIGHST(ipoint)'
WRITE(IFIL14,'(A)')' N $ are there smeared rings?'
WRITE(IFIL14,'(A)')' N $ print Cij?'
WRITE(IFIL14,'(A)')' N $ print loads?'
C
C end of Segment iseg input data
IALL = IALL + NMESH(iseg) + 2
2000 continue
C
C Begin Segment nseg+1 data (cylindrical segment)
C
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IF (LENCYL.GT.0.001) THEN
NMESH(nseg+1) = 51
WRITE(IFIL14,'(I4,A)') NMESH(nseg+1), ' $ NMESH seg.nseg+1'
WRITE(IFIL14,'(A)')' 1 $ NTYPEH'
WRITE(IFIL14,'(A)')' 4 $ NHVALU'

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WRITE(IFIL14,'(A)')' 1 $ IHVALU'
WRITE(IFIL14,'(A)')' 25 $ IHVALU'
WRITE(IFIL14,'(A)')' 26 $ IHVALU'
WRITE(IFIL14,'(A)')' 50 $ IHVALU'
WRITE(IFIL14,'(A)')' 0.2 $ HVALU'
WRITE(IFIL14,'(A)')' 0.2 $ HVALU'
WRITE(IFIL14,'(A)')' 1.0 $ HVALU'
WRITE(IFIL14,'(A)')' 1.0 $ HVALU'
WRITE(IFIL14,'(A)')' 1 $ NSHAPE'
WRITE(IFIL14,'(1P,E14.6,A)') x2(nseg), ' $ R1'
WRITE(IFIL14,'(1P,E14.6,A)') y2(nseg), ' $ Z1'
WRITE(IFIL14,'(1P,E14.6,A)') x2(nseg), ' $ R2'
WRITE(IFIL14,'(1P,E14.6,A)') y2(nseg)+LENCYL, ' $ Z2'
WRITE(IFIL14,'(I4,A)') IMPERF, ' $ IMP'
IF (IMPERF.EQ.1) THEN
    WRITE(IFIL14,'(A)')' 4 $ ITYPE'
    WRITE(IFIL14,'(1P,E14.6,A)') WIMP, ' $ WIMP'
    WRITE(IFIL14,'(A)')' 1 $ ISTART'
    NUMB = NMESH(nseg+1) + 2
    WRITE(IFIL14,'(I4,A)') NUMB, ' $ NUMB'
    DO 70 I = 1,NUMB
        J = I + IALL
        WRITE(IFIL14,'(1P,E14.6,A)') WMODEX(J), ' $ WSHAPE'
70    CONTINUE
    WRITE(IFIL14,'(A)')' N $ any more modes?'
ENDIF
WRITE(IFIL14,'(A)')' 3 $ NTYPEZ'
WRITE(IFIL14,'(A)')' 0. $ ZVAL'
WRITE(IFIL14,'(A)')' N $ print r(s)...?'
WRITE(IFIL14,'(A)')' 0 $ NRINGS'
WRITE(IFIL14,'(A)')' 0 $ K'
WRITE(IFIL14,'(A)')' 0 $ LINTYP'
WRITE(IFIL14,'(A)')' 1 $ IDISAB'
WRITE(IFIL14,'(A)')' 1 $ NLTYPE'
WRITE(IFIL14,'(A)')' 2 $ NPSTAT'
WRITE(IFIL14,'(A)')' 0 $ NLOAD(1)'
WRITE(IFIL14,'(A)')' 0 $ NLOAD(2)'
WRITE(IFIL14,'(A)')' 1 $ NLOAD(3)'
WRITE(IFIL14,'(A)')' 1. $ PN(1)'
WRITE(IFIL14,'(A)')' 1. $ PN(2)'
WRITE(IFIL14,'(A)')' 2 $ NTYPE'
WRITE(IFIL14,'(1P,E14.6,A)') y2(nseg), ' $ Z1'
WRITE(IFIL14,'(1P,E14.6,A)') y2(nseg)+LENCYL, ' $ Z2'
WRITE(IFIL14,'(A)')' 2 $ NWALL'
WRITE(IFIL14,'(1P,E14.6,A)') EMATL, ' $ E'
WRITE(IFIL14,'(1P,E14.6,A)') NUMATL, ' $ U'
WRITE(IFIL14,'(A)')' 0. $ SM'
WRITE(IFIL14,'(A)')' 0. $ ALPHA'

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WRITE(IFIL14,'(A)')' 0 $ NRS'
WRITE(IFIL14,'(A)')' -1 $ NSUR'
WRITE(IFIL14,'(A)')' 3 $ NTYPE'
WRITE(IFIL14,'(1P,E14.6,A)') THKCYL, ' $ TVAL'
WRITE(IFIL14,'(A)')' N $ print ref. surf?'
WRITE(IFIL14,'(A)')' N $ print Cij?'
WRITE(IFIL14,'(A)')' N $ print loads?'
ENDIF
C23456789012345678901234567890123456789012345678901234567890123456789012
C End of (LENCYL.GT.0.001)
C
C End of input for Segment nseg+1 (cylindrical segment)
C
C Start GLOBAL data..
C
WRITE(IFIL14,'(A)')' 1 $ NLAST'
WRITE(IFIL14,'(A)')' N $ expanded plots?'
C
C Following for linear buckling of perfect shell...
IF (INDX.EQ.1) THEN
WRITE(IFIL14,'(A)')' 0 $ NOB'
WRITE(IFIL14,'(A)')' 0 $ NMINB'
WRITE(IFIL14,'(A)')' 0 $ NMAXB'
WRITE(IFIL14,'(A)')' 1 $ INCRB'
WRITE(IFIL14,'(A)')' 10 $ NVEC'
WRITE(IFIL14,'(A)')' 0. $ P'
WRITE(IFIL14,'(1P,E14.6,A)') PRESS(ILOADX)/1000.0, ' $ DP'
WRITE(IFIL14,'(A)')' 0. $ TEMP'
WRITE(IFIL14,'(A)')' 0. $ DTEMP'
WRITE(IFIL14,'(A)')' 0. $ OMEGA'
WRITE(IFIL14,'(A)')' 0. $ DOMECA'
ENDIF
C
C23456789012345678901234567890123456789012345678901234567890123456789012
C Following is for nonlinear axisymmetric collapse...
IF (INDX.EQ.2) THEN
WRITE(IFIL14,'(1P,E14.6,A)') PMAX/10.0, ' $ P'
WRITE(IFIL14,'(1P,E14.6,A)') PMAX/10.0, ' $ DP'
WRITE(IFIL14,'(A)')' 0. $ TEMP'
WRITE(IFIL14,'(A)')' 0. $ DTEMP'
WRITE(IFIL14,'(A)')' 20 $ NSTEPS'
WRITE(IFIL14,'(A)')' 0. $ OMEGA'
WRITE(IFIL14,'(A)')' 0. $ DOMECA'
ENDIF
C
C Following is for nonlinear non-axisymmetric bifurcation buckling
C of imperfect shell...
IF (INDX.EQ.3) THEN

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        WRITE(IFIL14,'(I4,A)') NOBX, ' $ NOB'
        WRITE(IFIL14,'(I4,A)') NMINBX, ' $ NMINB'
        WRITE(IFIL14,'(I4,A)') NMAXBX, ' $ NMAXB'
        WRITE(IFIL14,'(I4,A)') INCRBX, ' $ INCRB'
        WRITE(IFIL14,'(A)') 1 $ NVEC'
        WRITE(IFIL14,'(1P,E14.6,A)') PMAX , ' $ P'
C BEG MAR 2008
    IF (INDIC.NE.-2)
1    WRITE(IFIL14,'(1P,E14.6,A)') PMAX/1000.0, ' $ DP'
    IF (INDIC.EQ.-2)
1    WRITE(IFIL14,'(1P,E14.6,A)') PMAX/100.0, ' $ DP'
C END MAR 2000
    WRITE(IFIL14,'(A)') 0. $ TEMP'
    WRITE(IFIL14,'(A)') 0. $ DTEMP'
C BEG MAR 2008
    IF (INDIC.EQ.-2)
1    WRITE(IFIL14,'(A)') 50 $ Number of steps'
C END MAR 2008
    WRITE(IFIL14,'(A)') 0. $ OMEGA'
    WRITE(IFIL14,'(A)') 0. $ DOME GA'
ENDIF
C
C Following is for nonlinear axisymmetric stress analysis...
    IF (INDX.EQ.4) THEN
        WRITE(IFIL14,'(1P,E14.6,A)') PMAX/10.0, ' $ P'
        WRITE(IFIL14,'(1P,E14.6,A)') PMAX/10.0, ' $ DP'
        WRITE(IFIL14,'(A)') 0. $ TEMP'
        WRITE(IFIL14,'(A)') 0. $ DTEMP'
        WRITE(IFIL14,'(A)') 10 $ NSTEPS'
        WRITE(IFIL14,'(A)') 0. $ OMEGA'
        WRITE(IFIL14,'(A)') 0. $ DOME GA'
    ENDIF
C
C Start CONSTRAINTS...
C
    IF (LENCYL.GT.0.001)
1    WRITE(IFIL14,'(I4,A)') nseg+1, ' $ nseg'
    IF (LENCYL.LE.0.001)
1    WRITE(IFIL14,'(I4,A)') nseg, ' $ nseg'
C
    Do 3000 iseg = 1,nseg
C
        if (iseg.eq.1) then
C Segment 1 constraint pole condition...
            WRITE(IFIL14,'(A)') 1 $ number of poles'
            WRITE(IFIL14,'(A)') 1 $ nodal point at pole'
            WRITE(IFIL14,'(A)') 0 $ grounded how many stations?'
            WRITE(IFIL14,'(A)') N $ joined to lower segs?

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endif
C234567890123456789012345678901234567890123456789012345678901234567890123456789012
if (iseg.eq.nseg) then
C Segment nseg constraint conditions...
WRITE(IFIL14,'(A)')' 0 $ number of poles'
IF (LENCYL.GT.0.001)
1 WRITE(IFIL14,'(A)')' 0 $ grounded how many stations?'
IF (LENCYL.LE.0.001) THEN
WRITE(IFIL14,'(A)')' 1 $ grounded how many stations?'
WRITE(IFIL14,'(I4,A)') NMESH(nseg),' $ INODE = node'
WRITE(IFIL14,'(A)')' 1 $ IUSTAR constrained'
WRITE(IFIL14,'(A)')' 1 $ IVSTAR constrained'
WRITE(IFIL14,'(A)')' 0 $ IWSTAR constrained'
WRITE(IFIL14,'(A)')' 1 $ ICHI constrained'
WRITE(IFIL14,'(A)')' 0. $ D1=radial eccentricity'
WRITE(IFIL14,'(A)')' 0. $ D2=axial eccentricity'
WRITE(IFIL14,'(A)')' N $ bc same prebuck & buck.?'
WRITE(IFIL14,'(A)')' 1 $ IUSTARB constrained'
WRITE(IFIL14,'(A)')' 1 $ IVSTARB constrained'
WRITE(IFIL14,'(A)')' 0 $ IWSTARB constrained'
WRITE(IFIL14,'(A)')' 1 $ ICHIB constrained'
ENDIF
C End of (LENCYL.LE.0.001) condition
endif
C234567890123456789012345678901234567890123456789012345678901234567890123456789012
if (iseg.gt.1) then
if (iseg.lt.nseg) then
WRITE(IFIL14,'(A)')' 0 $ number of poles'
WRITE(IFIL14,'(A)')' 0 $ grounded how many stations?'
endif
WRITE(IFIL14,'(A)')' Y $ joined to lower segs?'
WRITE(IFIL14,'(A)')' 1 $ at how many stations joined?'
WRITE(IFIL14,'(A)')' 1 $ INODE= node of current seg.'
WRITE(IFIL14,'(I4,A)') iseg-1,' $ JSEG=previous segment'
WRITE(IFIL14,'(I4,A)') NMESH(iseg-1),' $ JNODE prev.seg.'
WRITE(IFIL14,'(A)')' 1 $ IUSTAR constrained'
WRITE(IFIL14,'(A)')' 1 $ IVSTAR constrained'
WRITE(IFIL14,'(A)')' 1 $ IWSTAR constrained'
WRITE(IFIL14,'(A)')' 1 $ ICHI constrained'
WRITE(IFIL14,'(A)')' 0. $ D1=radial eccentricity'
WRITE(IFIL14,'(A)')' 0. $ D2=axial eccentricity'
WRITE(IFIL14,'(A)')' Y $ bc same for prebuck & buck.?'
endif
C
3000 continue
C
C234567890123456789012345678901234567890123456789012345678901234567890123456789012
C

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      IF (LENCYL.GT.0.001) THEN
C Segment nseg+1 constraint conditions...
      WRITE(IFIL14,'(A)')' 0           $ number of poles'
      WRITE(IFIL14,'(A)')' 2           $ grounded at how many stations?'
      WRITE(IFIL14,'(A)')' 1           $ INODE= node of current seg.'
      WRITE(IFIL14,'(A)')' 1           $ IUSTAR constrained'
      WRITE(IFIL14,'(A)')' 0           $ IVSTAR constrained'
      WRITE(IFIL14,'(A)')' 0           $ IWSTAR constrained'
      WRITE(IFIL14,'(A)')' 0           $ ICHI   constrained'
      WRITE(IFIL14,'(A)')' 0.          $ D1=radial eccentricity'
      WRITE(IFIL14,'(A)')' 0.          $ D2=axial  eccentricity'
      WRITE(IFIL14,'(A)')' N           $ bc same for prebuck & buck.?'
      WRITE(IFIL14,'(A)')' 0           $ IUSTARB constrained'
      WRITE(IFIL14,'(A)')' 0           $ IVSTARB constrained'
      WRITE(IFIL14,'(A)')' 0           $ IWSTARB constrained'
      WRITE(IFIL14,'(A)')' 0           $ ICHIB   constrained'
      WRITE(IFIL14,'(I4,A)') NMESH(nseg+1),' $ INODE= node of constr'
      WRITE(IFIL14,'(A)')' 1           $ IUSTAR constrained'
      WRITE(IFIL14,'(A)')' 1           $ IVSTAR constrained'
      WRITE(IFIL14,'(A)')' 0           $ IWSTAR constrained'
      WRITE(IFIL14,'(A)')' 1           $ ICHI   constrained'
      WRITE(IFIL14,'(A)')' 0.          $ D1=radial eccentricity'
      WRITE(IFIL14,'(A)')' 0.          $ D2=axial  eccentricity'
      WRITE(IFIL14,'(A)')' N           $ bc same for prebuck & buck.?'
      WRITE(IFIL14,'(A)')' 1           $ IUSTARB constrained'
      WRITE(IFIL14,'(A)')' 1           $ IVSTARB constrained'
      WRITE(IFIL14,'(A)')' 1           $ IWSTARB constrained'
      WRITE(IFIL14,'(A)')' 1           $ ICHIB   constrained'
      WRITE(IFIL14,'(A)')' Y           $ joined to lower segs?'
      WRITE(IFIL14,'(A)')' 1           $ at how many stations joined?'
      WRITE(IFIL14,'(A)')' 1           $ INODE= node of current seg.'
      WRITE(IFIL14,'(A)')' 2           $ JSEG = previous segment'
      WRITE(IFIL14,'(I4,A)') NMESH(nseg),' $ JNODE=node prev. seg.'
      WRITE(IFIL14,'(A)')' 1           $ IUSTAR constrained'
      WRITE(IFIL14,'(A)')' 1           $ IVSTAR constrained'
      WRITE(IFIL14,'(A)')' 1           $ IWSTAR constrained'
      WRITE(IFIL14,'(A)')' 1           $ ICHI   constrained'
      WRITE(IFIL14,'(A)')' 0.          $ D1=radial eccentricity'
      WRITE(IFIL14,'(A)')' 0.          $ D2=axial  eccentricity'
      WRITE(IFIL14,'(A)')' Y           $ bc same for prebuck & buck.?'
      ENDIF
C      End of (LENCYL.GT.0.001) condition
C
      WRITE(IFIL14,'(A)')' N           $ rigid body possible?'
C23456789012345678901234567890123456789012345678901234567890123456789012
      IF (INDX.EQ.4) THEN
        do 3010 iseg = 1,nseg
          WRITE(IFIL14,'(A)')' Y           $ output for seg. i?'

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```

3010    continue
        IF (LENCYL.GT.0.001)
1      WRITE(IFIL14,'(A)')'  N          $ output for seg. nseg+1?'
        WRITE(IFIL14,'(A)')'  Y          $ output for rings?'
    ELSE
        do 3020 iseg = 1,nseg
        WRITE(IFIL14,'(A)')'  Y          $ output for seg. i?'
3020    continue
        IF (LENCYL.GT.0.001)
1      WRITE(IFIL14,'(A)')'  Y          $ output for seg. nseg+1?'
        WRITE(IFIL14,'(A)')'  Y          $ output for rings?'
    ENDIF
C
    RETURN
    END

c
c
c
C=DECK      x3y3
    SUBROUTINE x3y3(ifile,iseg,x1,y1,x2,y2,x03,y03,x3,y3)
c  input:
c  (x1,y1), (x2,y2) = end points that lie on the original ellipse
c  (x03,y03) = point where normals to the ellipse at (x1,y1) and
c              (x2,y2) intersect
c  output:
c  (x3,y3) center of curvature of the "equivalent" toroidal segment.
c
c  (x3,y3) are determined by Newton's method from two nonlinear
c  equations in dx,dy, in which dx,dy are the distances between
c  x03,y03 and x3,y3.
c
    double precision x1,y1,x2,y2,x3,y3,x03,y03
    double precision d1sq,d2sq,delsq,a1,a2,b1,b2
    double precision f1,f1p,f1pp, f2,f2p,f2pp
    double precision dx,dy,u,v
c
c  For a toroidal segment, the two distances from (x3,y3) to the two
c  segment end points (x1,y1) and (x2,y2) must be equal. In other
c  words the meridional radius of curvature of the torioidal segment
c  must be constant in that segment.
c
c  However, in the ellipse these two distances are different. The
c  square of the difference is given by delta**2 (delsq):
c
    d1sq = (x1 - x03)**2 + (y1 - y03)**2
    d2sq = (x2 - x03)**2 + (y2 - y03)**2
    delsq = d1sq - d2sq
c

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```

c Here we determine the location of the center of meridional
c curvature of the "equivalent" torioidal segment by allocating
c half of delsq to each (distance)**2, d1sq and d2sq. We have two
c (distances)**2 that are equal:
c
c   (x1 - x03)**2 + (y1 - y03)**2 - delsq/2
c   (x2 - x03)**2 + (y2 - y03)**2 + delsq/2
c
c We must solve the following two nonlinear equations for (dx,dy):
c
c [x1 - (x03+dx)]**2 + [y1 - (y03+dy)]**2 =
c                               (x1 - x03)**2 + (y1 - y03)**2 - delsq/2   (1)
c
c [x2 - (x03+dx)]**2 + [y2 - (y03+dy)]**2 =
c                               (x2 - x03)**2 + (y2 - y03)**2 + delsq/2   (2)
c
c We use Newton's method:
c
c For the ith Newton iteration, let
c
c dx(i) = dx(i-1) + u
c dy(i) = dy(i-1) + v
c
c Then we develop two linear equations for u and v for the ith iteration:
c
c u*(x03-x1+dx(i-1)) +v*(y03-y1 +dy(i-1)) = f1pp
c u*(x03-x2+dx(i-1)) +v*(y03-y2 +dy(i-1)) = f2pp
c
c solve them, add u and v to dx(i-1) and dy(i-1), respectively, and
c iterate. We keep iterating until convergence is achieved.
c
c   iter = 0
c   dx = 0.
c   dy = 0.
c
c 10 continue
c   iter = iter + 1
c
c   a1 = 2.*(x03 - x1 + dx)
c   a2 = 2.*(x03 - x2 + dx)
c   b1 = 2.*(y03 - y1 + dy)
c   b2 = 2.*(y03 - y2 + dy)
c
c   f1 = (x1 - x03)**2 + (y1 - y03)**2 - delsq/2.
c   f2 = (x2 - x03)**2 + (y2 - y03)**2 + delsq/2.
c   f1p = f1 - x1**2 + 2.*x1*x03 - x03**2
c   1    -y1**2 + 2.*y1*y03 - y03**2
c   f2p = f2 - x2**2 + 2.*x2*x03 - x03**2

```



```

1          -y2**2 + 2.*y2*y03 - y03**2
f1pp = f1p - dx*2.*(x03-x1) -dy*2.*(y03-y1) -dx**2 -dy**2
f2pp = f2p - dx*2.*(x03-x2) -dy*2.*(y03-y2) -dx**2 -dy**2
c
u = (b2*f1pp - b1*f2pp)/(b2*a1 - b1*a2)
v = (a2*f1pp - a1*f2pp)/(a2*b1 - a1*b2)
dx = dx + u
dy = dy + v
c
C23456789012345678901234567890123456789012345678901234567890123456789012
c    if (iter.eq.1) write(ifile,'(/,A,i3,/,A,A)')
c    1' ***** Results from Newton iterations for segment no.',iseg,
c    1' iter      x03      dx      y03      dy      u',
c    1'          v'
c    write(ifile,'(i3,1p,6e12.4)')
c    1 iter, x03, dx, y03, dy, u, v
c
c    if (iter.gt.100) then
c        write(ifile,'(A)')' No convergence.'
c        call exit
c    endif
c
c    if (iter.lt.3) go to 10
c    if (abs(u).gt.0.001*abs(dx)) go to 10
c    if (abs(v).gt.0.001*abs(dy)) go to 10
c
c    Convergence has been achieved
c
c    x3 = x03 + dx
c    y3 = y03 + dy
c
c    return
c    end
=====

```