Kdp p hu#Fddfxddwlrqv#

For these calculations it is assumed that the main air tank is equipped with a high-delivery regulator and that the hammer cycle consists of applying the output of this regulator to the air cylinder for a preset fraction of the cylinder's stroke.

The force exerted on the piston is constant for the fraction f of the stroke. Then the force on the piston declines according to the laws of thermodynamics for an adiabatic expansion.

The drives system is a cylinder with a piston attached to chain wrapped around a sprocket on the hammer axle.

F | daghu

$$A_{throw} \coloneqq \pi \cdot \left(\left(\frac{d_{throw}}{2} \right)^2 - \left(\frac{d_{rod}}{2} \right)^2 \right) = 26.507 \, \, in^2 \qquad Vol_{throw} \coloneqq A_{throw} \cdot stroke$$

P hfkdqlvp

$$J_{hammer} \coloneqq m_{head} \cdot len_{arm}^{\ \ 2} + m_{arm} \cdot \left(rac{len_{arm}}{2}
ight)^2$$
 rotational inertia of the arm+hammer

Wkhup rg | qdp Ifv

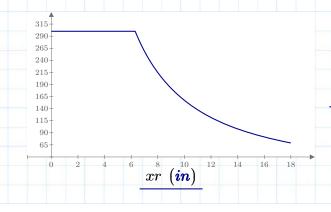
$$const(ff) \coloneqq P_0 \cdot (ff \cdot Vol_{throw})^{1.4}$$
 this is the constant required for the adiabatic expansion calc, as a function of fill fraction ff

$$V_{stroke}(x) := A_{throw} \cdot x$$

volume expression for the stroke

$$P\!\left(x,ff\right) \coloneqq \mathbf{if}\!\left(\!\left(\!\frac{V_{stroke}\!\left(x\right)}{Vol_{throw}}\!\leq\!ff\right)\!,\!P_{0},\!\frac{const\left(ff\right)}{V_{stroke}\!\left(x\right)^{1.4}}\!\right) \text{ the complete expression for the pressure } \left(\!\frac{V_{stroke}\!\left(x\right)}{Vol_{throw}}\!\right) = \mathbf{if}\!\left(\!\frac{V_{stroke}\!\left(x\right)}{Vol_{throw}}\!\right) + \mathbf{if}\!\left($$

 $xr \coloneqq 0 \cdot in, .1 \cdot in ... stroke$



P(xr,ffd) (psi)

Prwirq
$$r_{sprocket} \coloneqq \frac{stroke}{\theta_{max}} = 3.82 \; \textbf{in}$$

$$x_{cyl}(\theta) \coloneqq \theta \cdot r_{sprocket} \qquad x_{cyl}(180 \cdot \textbf{deg}) = 12 \; \textbf{in}$$

$$\theta_{hammer}(x) \coloneqq \frac{x}{r_{sprocket}} \qquad \theta_{stroke} \coloneqq \theta_{hammer}(stroke) = 270 \; \textbf{deg}$$

$$T_{axle}(\theta, ff) \coloneqq P(x_{cyl}(\theta), ff) \cdot A_{throw} \cdot r_{sprocket} \qquad \text{wrutxh} \text{#lisschg} \text{#e} | \text{#f} | \text{dqghu} \text{#rq} \text{#fudqn}$$

$$x_{cyl} \big(180 \cdot \mathbf{deg} \big) = 12 \, \, \mathbf{in}$$

$$heta_{hammer}(x) \coloneqq \frac{x}{r_{sprocket}} \qquad \qquad heta_{stroke} \coloneqq heta_{hammer}(stroke) = 270 \,\, oldsymbol{deg}$$

$$T_{axle}\big(\theta\,,ff\big)\coloneqq P\left(x_{cyl}\big(\theta\big)\,,ff\right)\cdot A_{throw}\cdot r_{sprocket} \quad \text{ which this soft give } |\,\#\,|\,\deg hu \# r \# hu + r$$

$$lpha_{hammer}ig(heta\,,\!ffig)\!\coloneqq\!rac{T_{axle}ig(heta\,,\!ffig)}{J_{hammer}}$$
 d

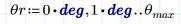
$$E_{hammer}(\theta,ff) \coloneqq \int_{0}^{\theta} T_{axle}(\theta,ff) d\theta$$

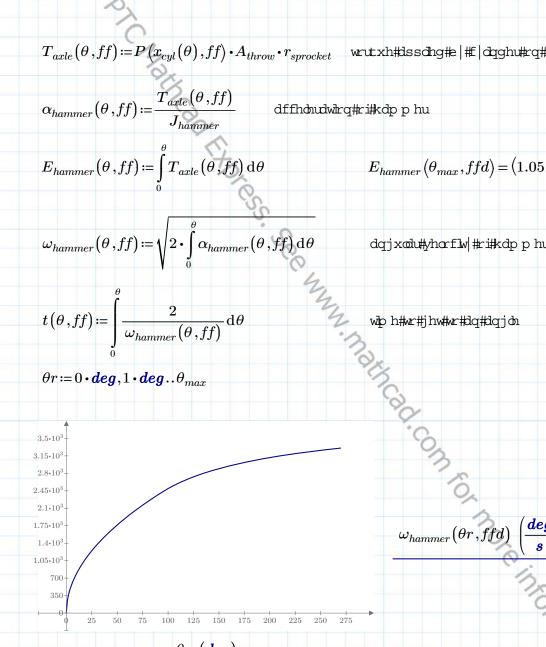
 $E_{hammer}(\theta_{max}, ffd) = (1.051 \cdot 10^4) J$

$$\omega_{hammer} (\theta, ff) \coloneqq \sqrt{2 \cdot \int_{0}^{\theta} \alpha_{hammer} (\theta, ff) \, \mathrm{d}\theta}$$

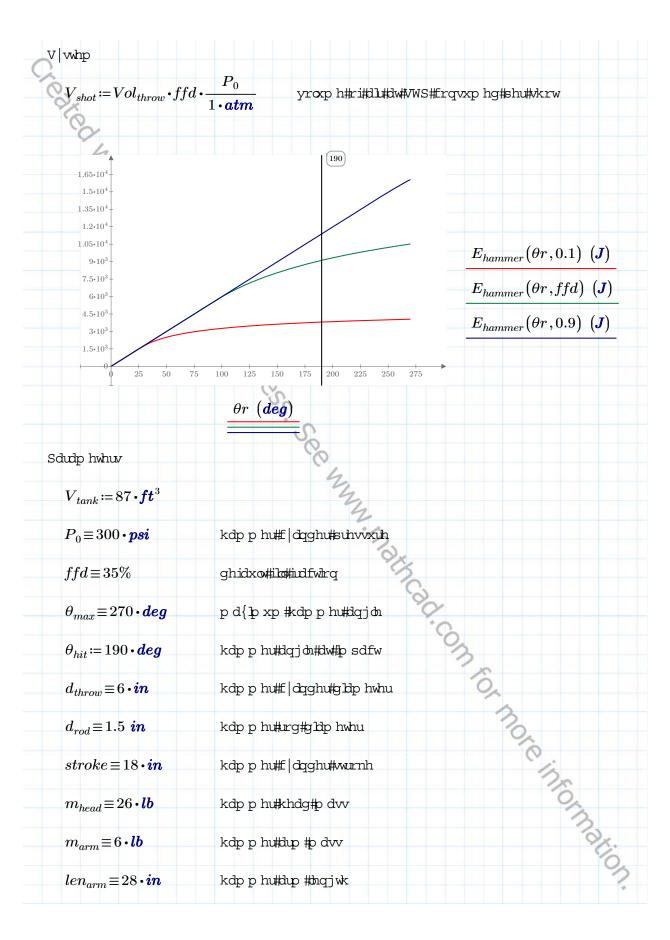
daj xodu#yharflw #ri#kdp p hu

$$t\left(heta\,,ff
ight)\!\coloneqq\!\int\limits_{0}^{ heta}rac{2}{\omega_{hammer}ig(heta\,,ffig)}\mathrm{d} heta$$





 $\theta r \; (deg)$



$$r_{sprocket}\!=\!3.82$$
 in

$$t\left(\theta_{hit},ffd\right) = 0.224 \ \boldsymbol{s}$$

vz lqj#wlph

$$E_{hammer}(\theta_{hit}, ffd) = 9109.5 \, \boldsymbol{J}$$

whup lqddwz lqj#hqhuj |

$$t_{chain} := A_{throw} \cdot P_0 = \left(7.952 \cdot 10^3\right) \ \textit{lbf}$$

$$w_{max}\!\coloneqq\!\omega_{hammer}\!\left(180\boldsymbol{\cdot}\boldsymbol{deg}\,,\!f\!fd\right)\!=\!506.516\;\boldsymbol{rpm}$$

$$n_{shots} = \frac{V_{tank}}{V_{shot}} = 44.1$$

$$F_{max} \coloneqq P_0 \cdot A_{throw} = \left(7.952 \cdot 10^3\right) \ \textit{lbf}$$

shdn#irufh#irp #ilh#f |dgqhu

$$F_{hammer} \coloneqq rac{T_{axle}ig(0\,,ffdig)}{len_{arm}} = ig(1.085 \cdot 10^3ig) \; m{lbf}$$

$$T_{max} := T_{axle}(0, ffd) = (2.531 \cdot 10^3) ft \cdot lbf$$

irufh#dw#kdp p hu#khdq

shdn#wrutxh#rq#kdp p hu#d{dn

Uhwdfw#sxoh#frqvlghudwlrqv

$$d_{retract} \equiv 4$$
 in

 $d_{retract}\!\equiv\!4$ in $d_{retract.rod}\!\equiv\!0.75$ in

$$A_{retract} \coloneqq \boldsymbol{\pi} \cdot \left(\left(\frac{d_{retract}}{2} \right)^2 - \left(\frac{d_{rod}}{2} \right)^2 \right) = 10.799 \, \, \boldsymbol{in}^2$$

$$Ve_{retract} := A_{retract} \cdot \frac{stroke}{3}$$

yroxp h#ri#hwdfw#vgh#dw#4;3#ghjuhhv

$$T_{retract}(P) \coloneqq A_{retract} \cdot P \cdot r_{sprocket}$$

$$A_{retract} \coloneqq \pi \cdot \left(\left(\frac{d_{retract}}{2} \right)^2 - \left(\frac{d_{retract.rod}}{2} \right)^2 \right) = 12.125 \, \, in^2$$

$$Fmax_{retract} = P_0 \cdot A_{retract} = \left(3.637 \cdot 10^3\right) \ \textit{lbf}$$

$$Fmin_{retract} \coloneqq \frac{T_{retract} \left(40 \cdot psi \right)}{len_{arm}} = 58.929 \ \textit{lbf}$$

$$Q_{retract} := 14 \cdot cfm$$

iorz #wdwh#ri#wig | #ydoyh#iurp #P fP dwwhu#dw#6333svl

$$t(P) \coloneqq rac{P \cdot Ve_{retract}}{1 \cdot bar \cdot Q_{retract}}$$

wip h#wr#iko#wr#d#jkyhq#suhvvxuh

$$t(100 \cdot psi) = 1.108 \ s$$

Kdp p hu#d{dn#dqg#ehdulqj=

$$F_{perBearing} = \frac{F_{max}}{2} = (3.976 \cdot 10^3) \ lbf$$

$$d_{axle} \coloneqq 55$$
 $mm = 2.165$ in

$$l_{axle} \coloneqq 120 \ \textit{mm} = 4.724 \ \textit{in}$$

$$A_{effAxle}\coloneqq d_{axle} \cdot l_{axle} = 10.23$$
 in 2

$$P_{effAxle} := \frac{F_{perBearing}}{A_{offAxle}} = 388.668 \ \textit{psi}$$

$$\begin{split} P_{effAxle} &\coloneqq \frac{F_{perBearing}}{A_{effAxle}} = 388.668 \ \textbf{\textit{psi}} \\ v_{surf} &\coloneqq \omega_{hammer} \left(180 \cdot \textbf{\textit{deg}}, ffd\right) \cdot \frac{d_{axle}}{2} = 1.459 \ \frac{\textbf{\textit{m}}}{\textbf{\textit{s}}} \end{split}$$

 $\omega_{hammer} (180 \cdot deg, ffd) = 506.516 \frac{rpm}{}$

$$PV \coloneqq P_{effAxle} \cdot v_{surf} = \left(1.116 \cdot 10^5\right) \, rac{ft}{min} \cdot psi$$

Z kdwtwrtp doddydd hynutirutdqtdoop loop td{onB

$$\boldsymbol{\tau}_{y2_6061}\!\coloneqq\!30\boldsymbol{\cdot ksi}$$

$$au_{y1_6061}\!\coloneqq\!27$$
 ksi

$$A_{double} \coloneqq 2 \cdot \boldsymbol{\pi} \cdot \left(\frac{d_{axle}}{2}\right)^2 = 7.365 \, \, \boldsymbol{in}^2$$

$$P_{swing} \coloneqq rac{F_{max}}{A_{double}} = 1.08 \; extbf{ksi}$$
 Z rz /#wrwd σ #lth1

Irufh#rq#fkdlq#gohw#) #xssrw#wxfwwh=#

$$\theta_{idler} = 159.2 \cdot deg$$

$$egin{aligned} heta_{idler} &\coloneqq 159.2 \cdot oldsymbol{deg} \ F_{halfIdler} &\coloneqq F_{max} \cdot \cos \left(rac{ heta_{idler}}{2}
ight) = \left(1.436 \cdot 10^3
ight) \, oldsymbol{lbf} \end{aligned}$$

$$F_{idler} = 2 \cdot F_{halfIdler} = (2.871 \cdot 10^3) \ lbf$$

$$\sigma_{y1_4_20bolt}\!\coloneqq\!5730\!\cdot\! \mathit{lbf}$$
 shufferofflind

dffruglj#wr#Z hlshgld/#ww#dssur{p ldwhd #wxh#wkdw#

$$au_{y1_4_20bolt} = 0.58 \cdot \sigma_{y1_4_20bolt} = \left(3.323 \cdot 10^3\right) \; \textit{lbf}$$

$$egin{align*} & au_{y1_4_20bolt} \coloneqq 0.58 \cdot \sigma_{y1_4_20bolt} = \left(3.323 \cdot 1.58 \cdot \sigma_{y1_4_20bolt} \right) & au_{y1_4_20bolt} \coloneqq \frac{ au_{y1_4_20bolt}}{F_{halfIdler}} & au_{y1_4_20bolt} = 2.315 & au_{y1_4_20bolt} &$$

$$\sigma_{y6061} \coloneqq 42 \cdot ksi$$

$$\tau_{y6061} \coloneqq 0.55 \cdot \sigma_{y6061} = 23.1 \ ksi$$

$$e_{dist} = 1.492 \cdot ir$$

$$F_{tearout} \coloneqq au_{y6061} \cdot e_{dist} \cdot 0.25$$
 in $= \left(8.616 \cdot 10^3\right)$ lbf

$$SF_{tearout} := \frac{F_{tearout}}{F_{halfIdler}} = 6.002$$

Gulyh#grj#vkhdu#dgg#ehdulgj#ordgv

$$A_{shear.sprocket} \coloneqq 1.3 \ \emph{in}^2$$

$$A_{shear.shaft}\coloneqq 1.286~in^2$$

$$n_{dog.sprocket} \coloneqq 6$$

$$r_{dog.shaft} \coloneqq 1.286$$
 in

$$r_{dog.sprocket} \coloneqq 1.8$$
 in

$$F_{dog.sprocket} \coloneqq \frac{T_{axle} \! \left(0\,,\!f\!fd \right)}{r_{dog.sprocket}} \! = \! \left(1.688 \cdot 10^4 \right) \, \textit{lbf}$$

$$P_{shear.sprocket} \coloneqq \frac{F_{dog.sprocket}}{n_{dog.sprocket} \cdot A_{shear.sprocket}} = 2.163 \ \textit{ksi}$$

$$F_{dog.shaft} \coloneqq rac{T_{axle}ig(0\,,ffdig)}{r_{dog.shaft}} = ig(2.362 ullet 10^4ig) \; m{lbf}$$

$$P_{shear.shaft} \coloneqq \frac{F_{dog.shrocket} \cdot A_{shear.shaft}}{n_{dog.sprocket} \cdot A_{shear.shaft}} = 3.061 \text{ ksi}$$

$$V_{bearing.shaft} \coloneqq 1.5 \text{ in}$$

$$t_{shaft} \coloneqq 0.7 \text{ in} \qquad l_{shaft} \coloneqq 0.54 \text{ in}$$

$$F_{bearing.shaft} \coloneqq \frac{T_{ante}(0,ffd)}{r_{bearing.shaft}} = (2.025 \cdot 10^4) \text{ lbf}$$

$$P_{bearing.shaft} \coloneqq \frac{F_{bearing.shaft}}{n_{dog.sprocket} \cdot l_{shaft} \cdot l_{shaft}} = 8.929 \text{ ksi}$$

$$V_{bolt.sprocket} \coloneqq \frac{2.5}{2} \text{ in} = 1.25 \text{ in}$$

$$F_{bolt.sprocket} \coloneqq \frac{2.5}{2} \text{ in} = 1.25 \text{ in}$$

$$F_{bolt.sprocket} \coloneqq \frac{T_{ante}(0,ffd)}{r_{bolt.sprocket}} = (2.43 \cdot 10^4) \text{ lbf}$$

$$Irufhlishnikerow$$

$$d_{bolt.sprocket} \coloneqq \pi \cdot \left(\frac{d_{bolt.sprocket}}{2}\right)^2 = 0.049 \text{ in}^2$$

$$Duhdishnikerow$$

$$n_{bolt} \coloneqq 150 \text{ ksi}$$

$$F_{bolt.sprocket} \coloneqq \frac{F_{bolt.sprocket}}{n_{bolt} \cdot A_{bolt.sprocket}} = 41.253 \text{ ksi}$$

$$V_{bolt.sprocket} \coloneqq \frac{F_{bolt.sprocket}}{n_{bolt} \cdot A_{bolt.sprocket}} = 41.253 \text{ ksi}$$

$$V_{bolt.sprocket} \coloneqq \frac{F_{bolt.sprocket}}{n_{bolt} \cdot A_{bolt.sprocket}} = 41.253 \text{ ksi}$$

$$V_{bolt.hammer} \coloneqq \frac{F_{bolt.sprocket}}{n_{bolt.sprocket}} = 3.636$$

$$Idivardir illy dilling$$

$$F_{bolt.hammer} \coloneqq \frac{4.5}{2} \text{ in} = 2.25 \text{ in}$$

$$F_{bolt.hammer} \coloneqq \frac{T_{ante}(0,ffd)}{r_{bolt.hammer}} = (1.35 \cdot 10^4) \text{ lbf}$$