



# CS65K Robotics

Modelling, Planning and Control

## Chapter 5: Sensors and Actuators

LECTURE 10: SENSORS AND ACTUATORS

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# Objectives

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- Move forward from Manipulator to sensor and actuators.
- The model of a **servomotor** with power amplifier is derived
- **Velocity control ( $v$ )** vs **torque control ( $\omega$ )** of the electric drive system are presented
- The effects of mechanical transmission are analyzed
- The general scheme for control of an electric drive system is introduced

# Objectives

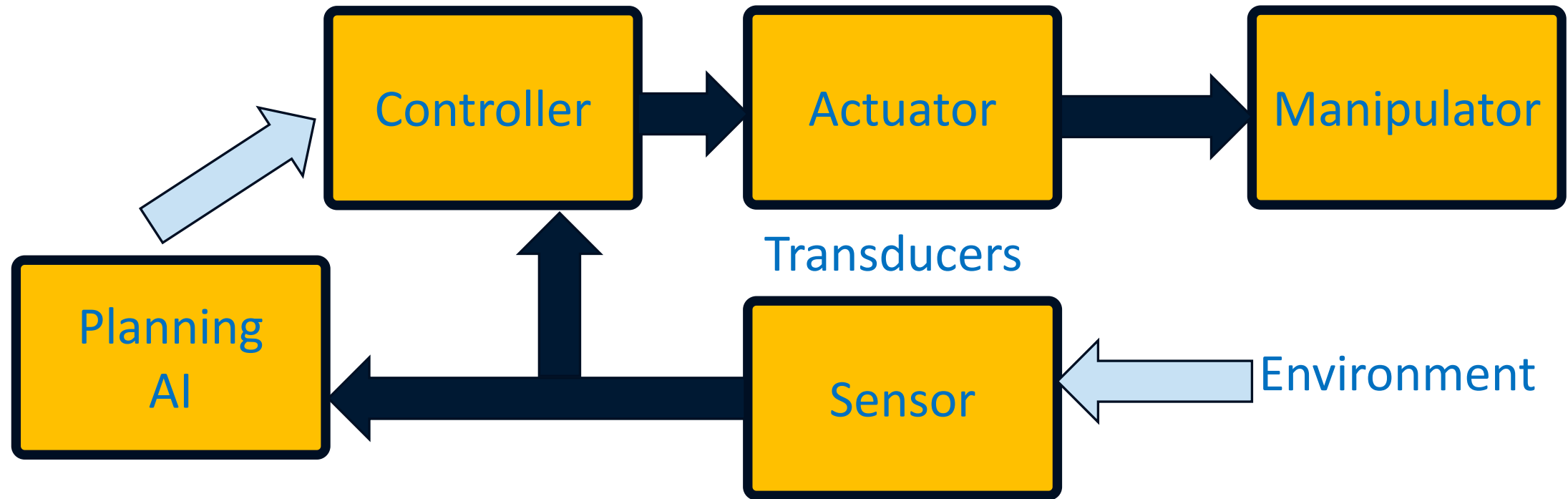
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- The motion control problem for robot manipulators is formulated
- The two techniques for joint space control are introduced
- The **dynamic model** is cast in a suitable form for decentralized control

# Overview

## SECTION 1

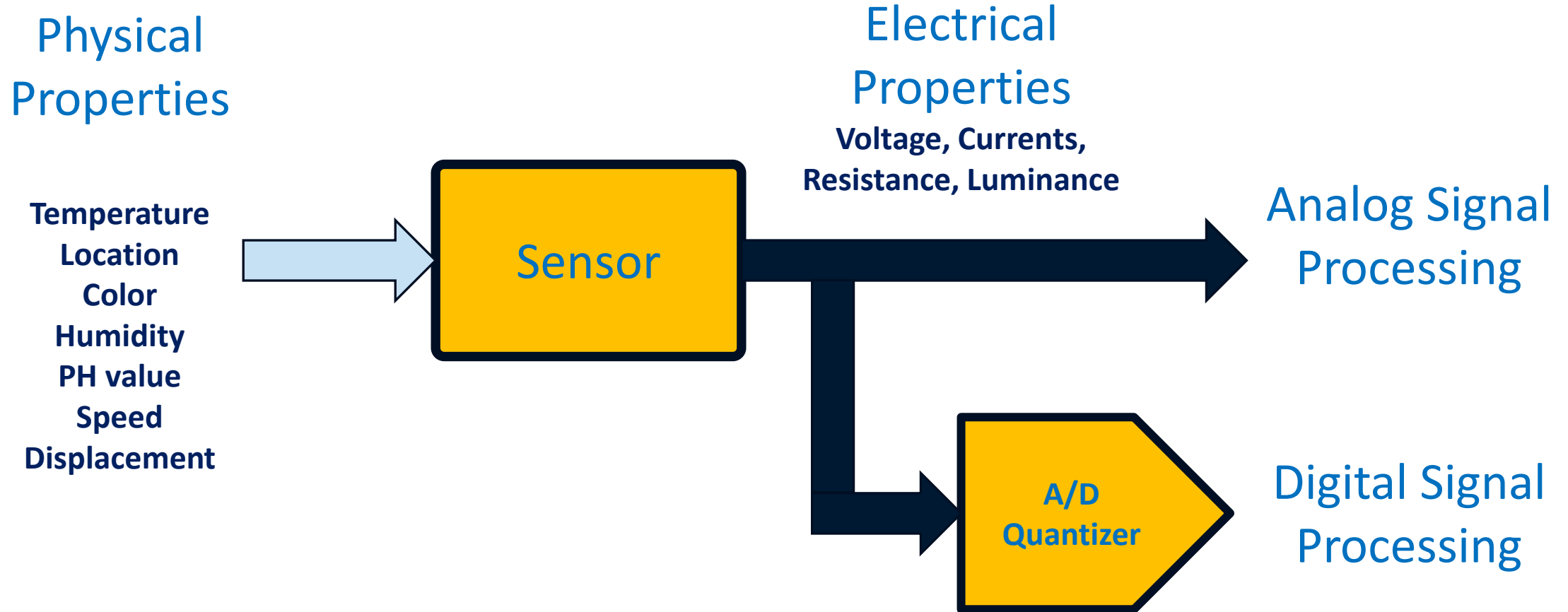
# Logical Units for A Robot



# Sensors

## SECTION 2

# Sensor



# Image Processing

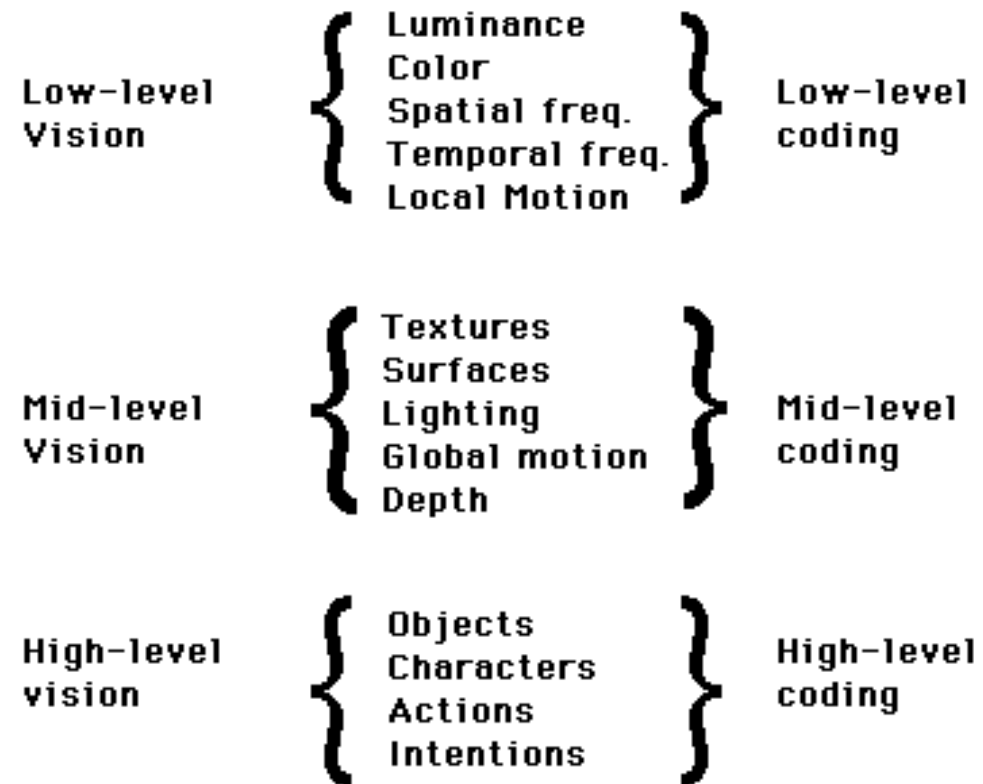
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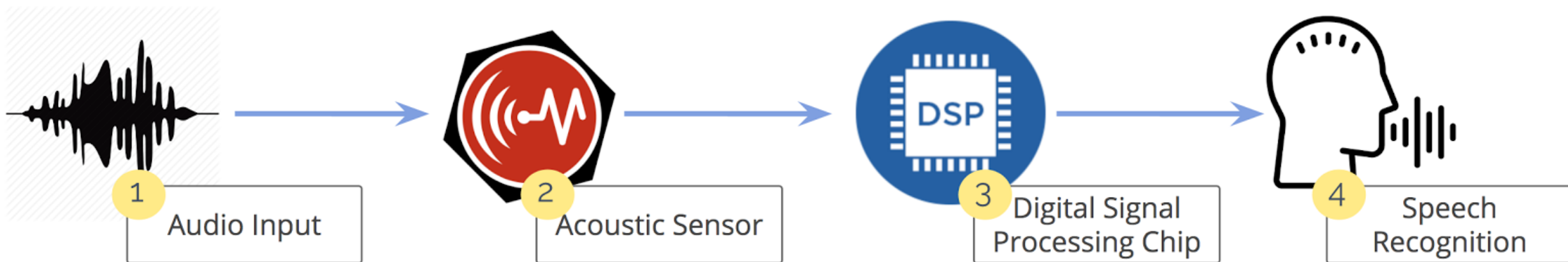
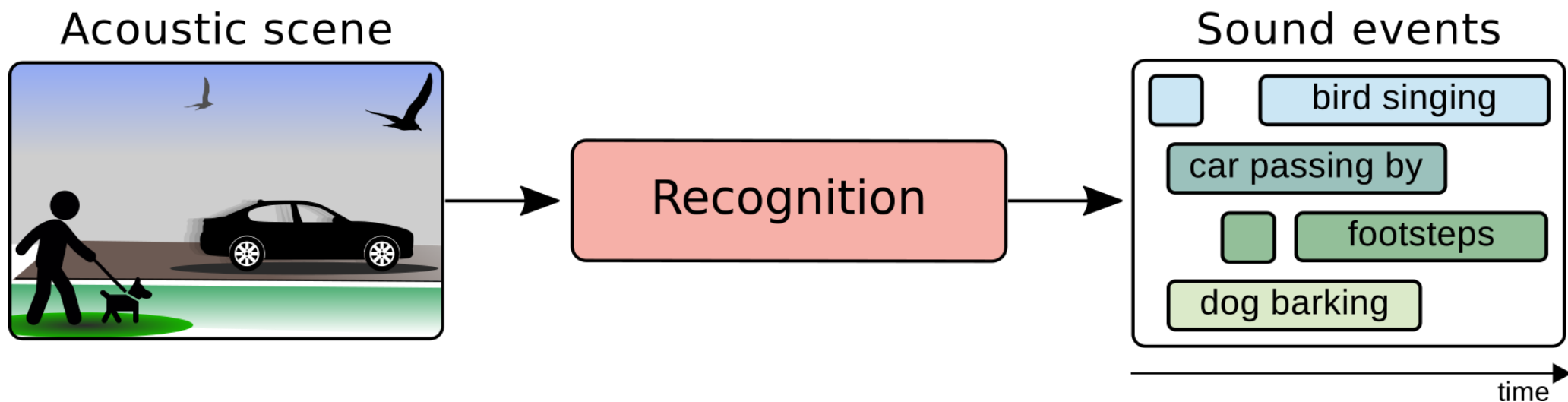
Type	Input	Output	Examples
Low Level Process	Image	Image	Noise removal, image sharpening
Mid-Level Process	Image	Attributes	Object recognition, Segmentation
High Level Process	Attributes	Understanding	Scene understanding, autonomous navigation



# Image Vision

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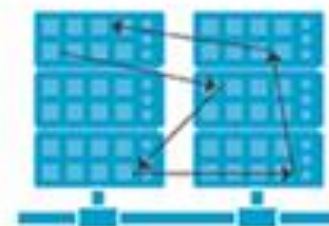
# Voice recognition



ANALOG AUDIO



ANALOG-TO-DIGITAL  
CONVERSION



PATTERN  
RECOGNITION



# Purpose of Sensors

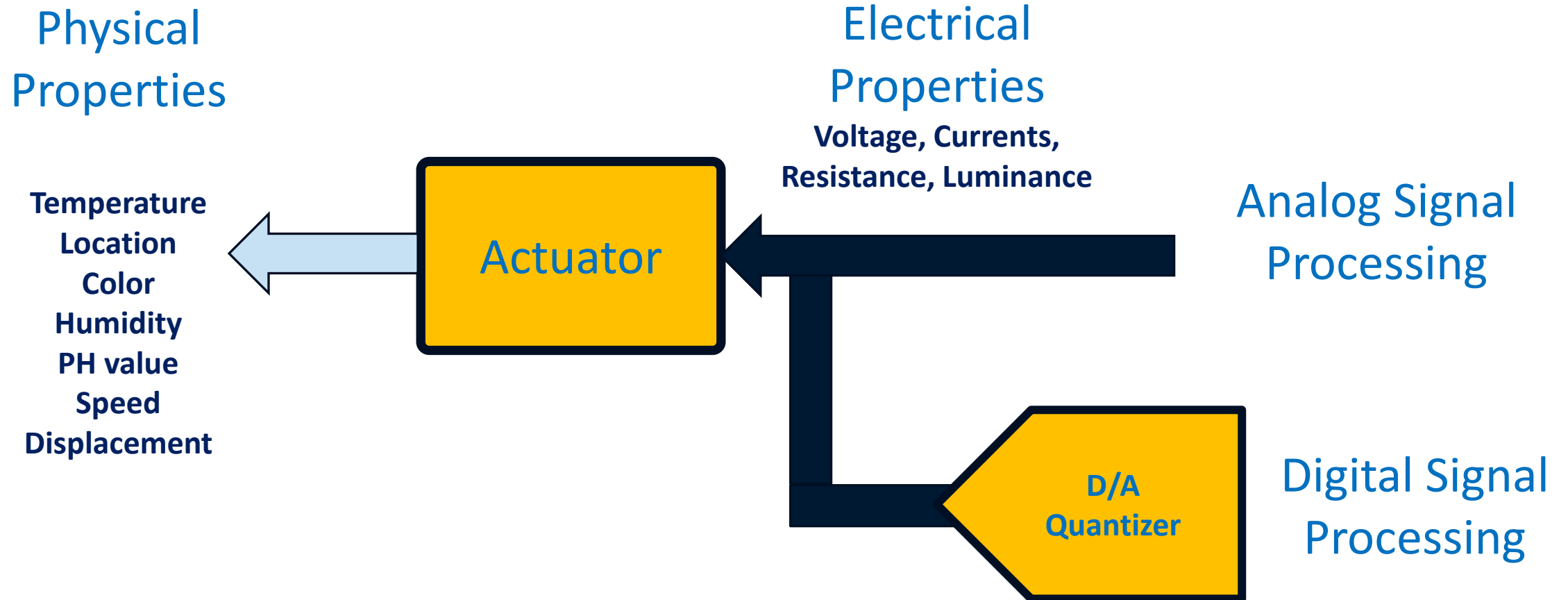
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- To give robot information about itself. (Joint angle, connection status)
- To give robot information about the environment.

# Actuators

## SECTION 3

# Actuators



# Transducers

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## Physical Properties

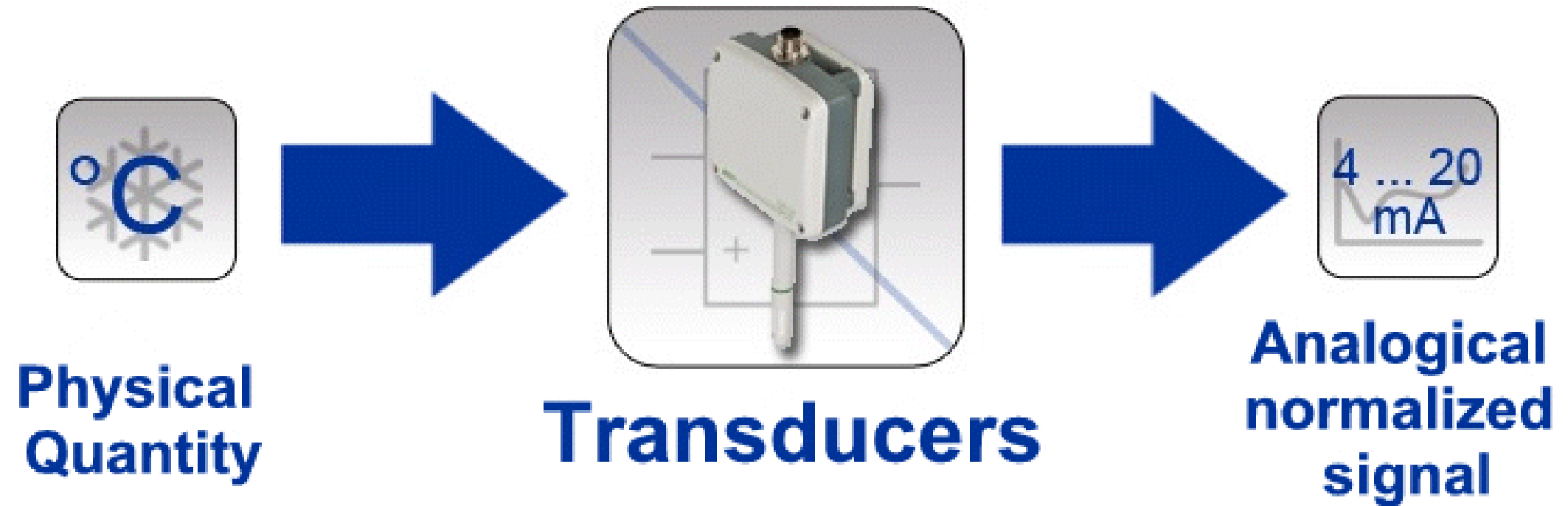
Temperature  
Location  
Color  
Humidity  
PH value  
Speed  
Displacement

## Transducers

Sensor

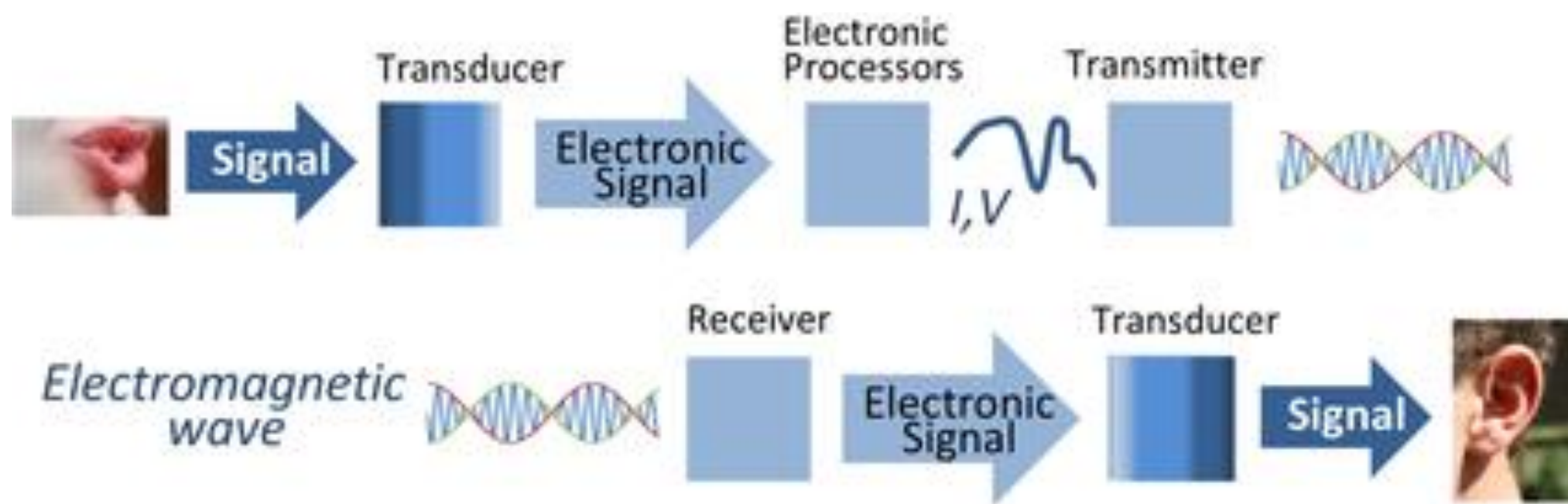
Actuator

Electrical  
Properties  
Voltage, Currents,  
Resistance, Luminance



Transducers





# End Effectors

- In [robotics](#), an end effector is the device or tool that's connected to the end of a robot arm and enables the robot arm to perform specific tasks
- Usually end effectors are custom engineered for a particular task



# End-to-End Model

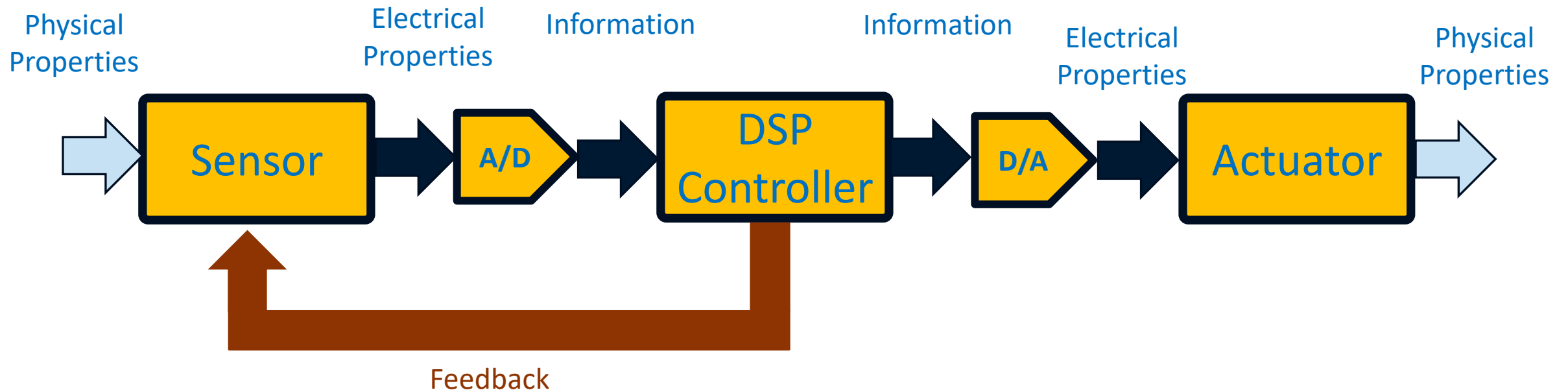
SECTION 4

# End-to-End Robot Model

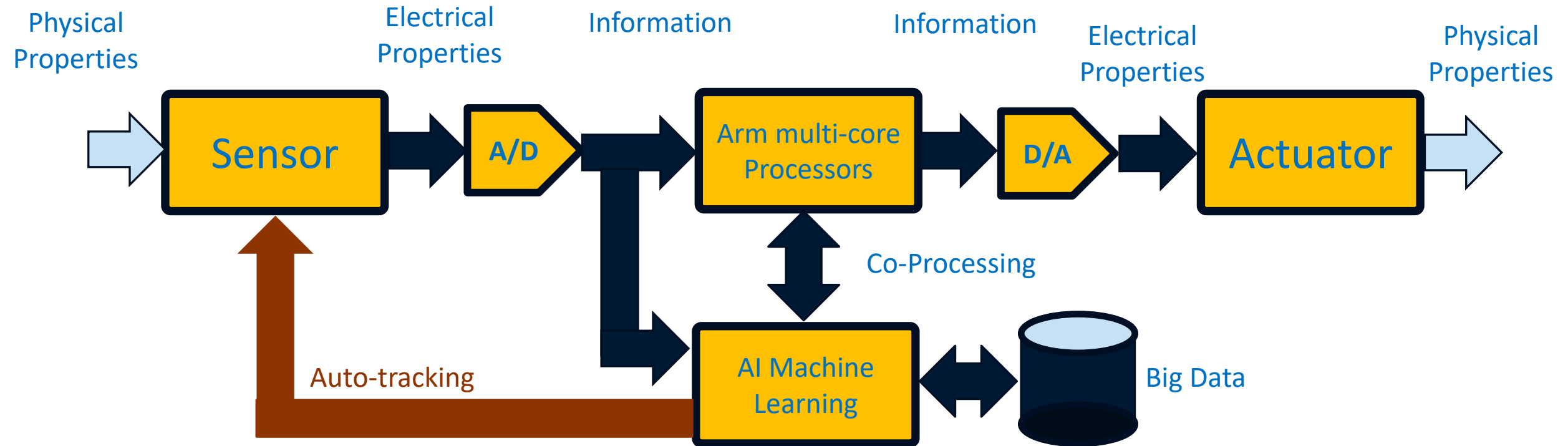
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# End-to-End DSP Robot Model



# End-to-End Smart Robot Model



# Model of Electric Drive System

SECTION 5

# Model of Electric Drive System

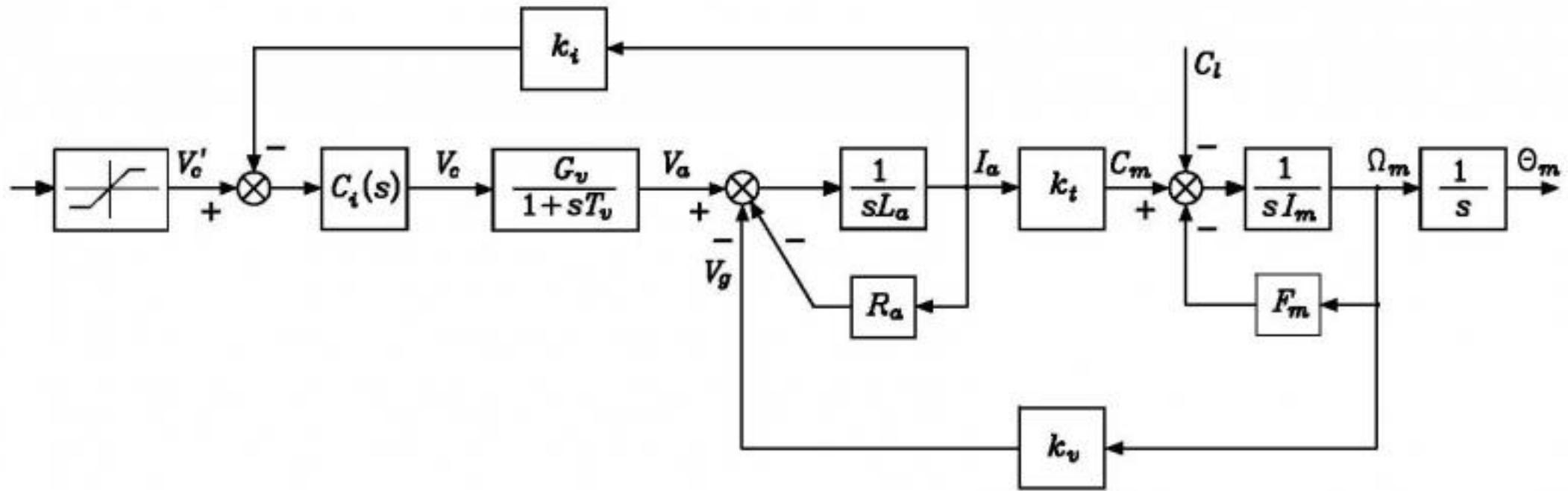
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From a modelling viewpoint, a permanent-magnet DC motor and a brushless DC motor provided with the commutation module and position sensor can be described by the same differential equations

## **Electric servomotor with amplifier**

- Electric balance 
$$V_a = (R_a + sL_a)I_a + V_g$$
$$V_g = k_v \Omega_m$$
- Mechanical balance 
$$C_m = (sI_m + F_m)\Omega_m + C_l$$
$$C_m = k_t I_a$$
- Power amplifier 
$$\frac{V_a}{V_c} = \frac{G_v}{1 + sT_v}$$
- Possibility of armature *current feedback*





**Block scheme of an electric drive**

# Velocity-Controlled Generator

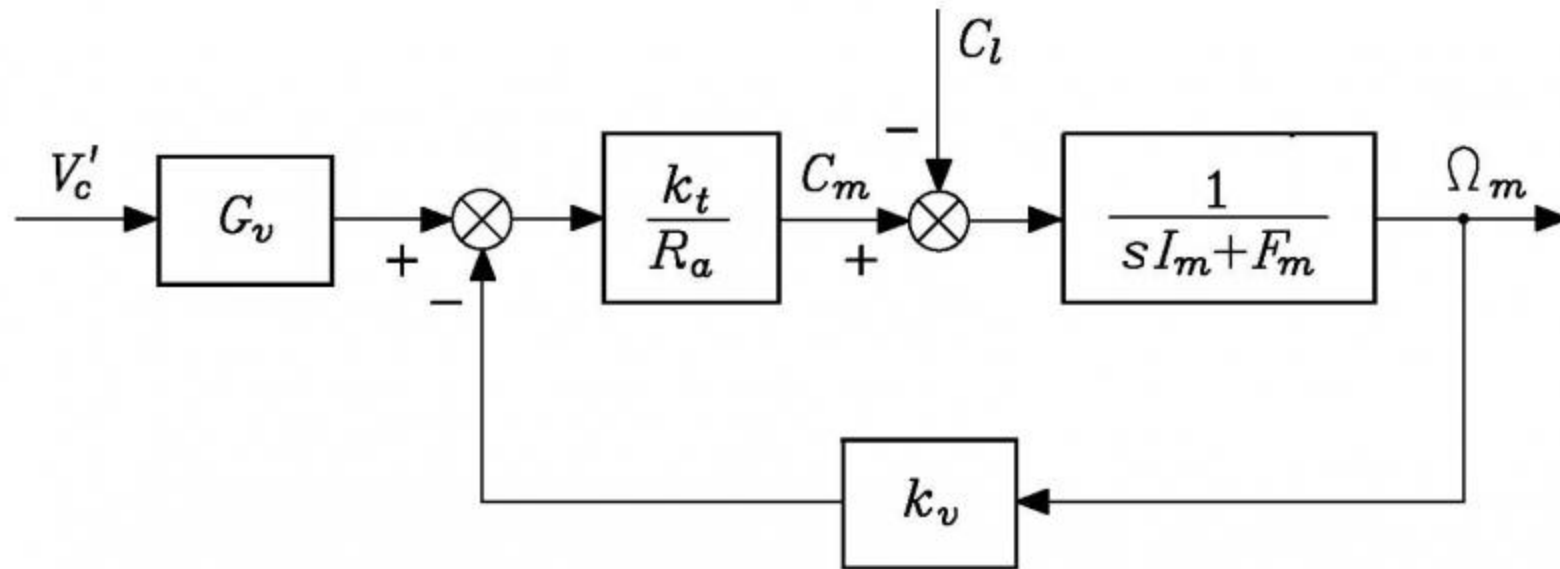
The choice of the regulator  $C_i(s)$  of the current loop allows a velocity-controlled or torque-controlled behaviour for the electric drive, depending on the values attained by the loop gain

- $k_i = 0$
- $F_m \ll \frac{k_v k_t}{R_a}$

$$\Omega_m = \frac{\frac{1}{k_v}}{1 + s \frac{R_a I_m}{k_v k_t}} G_v V'_c - \frac{\frac{R_a}{k_v k_t}}{1 + s \frac{R_a I_m}{k_v k_t}} C_l$$

- At steady state

$$\omega_m \approx \frac{G_v}{k_v} v'_c$$

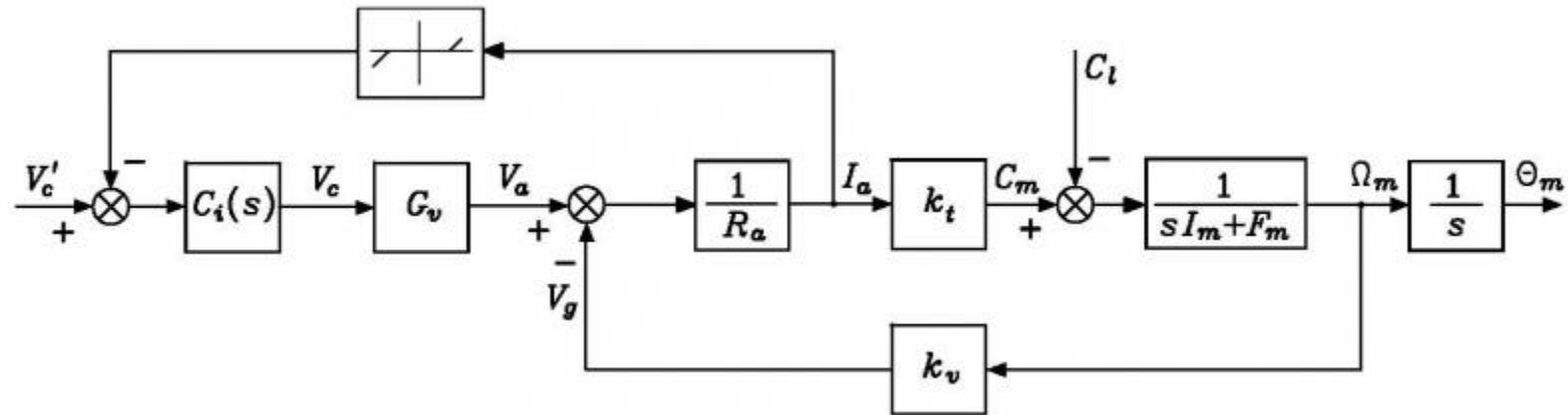


**Block scheme of an electric drive as a velocity-controlled generator**

# Current Protection

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- Setting a protection can be solved by introducing a current limit that is not performed by a saturation on the control signal but it exploits a current feedback with a dead-zone nonlinearity on the feedback path



**Block scheme of an electric drive with nonlinear current feedback**

# Torque-Controlled Generator

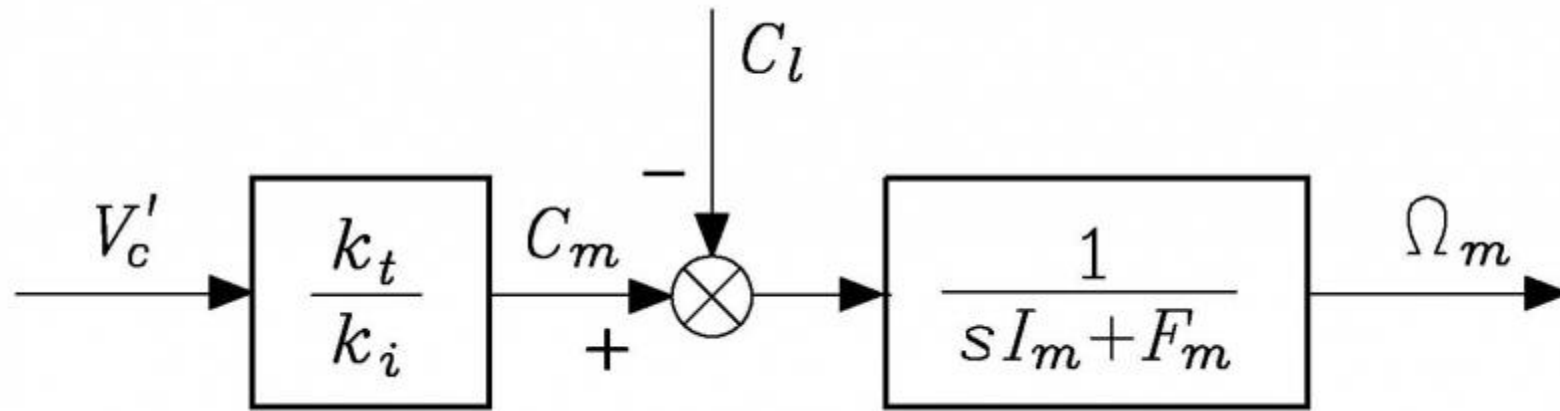
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- $Kk_i \gg R_a$
- $k_v\Omega/Kk_i \approx 0$

$$\Omega_m = \frac{\frac{k_t}{k_i F_m}}{1 + s \frac{I_m}{F_m}} V_c' - \frac{\frac{1}{F_m}}{1 + s \frac{I_m}{F_m}} C_l$$

- At steady state

$$c_m \approx \frac{k_t}{k_i} \left( v_c' - \frac{k_v}{G_v} \omega_m \right)$$



**Block scheme of an electric drive as a torque-controlled generator**

# Electric Drive Transfer Function

Relationship between the control input and the actuator position output

$$M(s) = \frac{k_m}{s(1 + sT_m)}$$

- Velocity-controlled generator  $k_m = \frac{1}{k_v} \quad T_m = \frac{R_a I_m}{k_v k_t}$
- Torque-controlled generator  $k_m = \frac{k_t}{k_i F_m} \quad T_m = \frac{I_m}{F_m}$

Without current feedback, the system has a better rejection of disturbance torques in terms of both equivalent

gain  $R_a/k_v k_t \ll 1/F_m$  and time response  $R_a I_m/k_v k_t \ll I_m/F_m$



# Transmission Effect

## SECTION 1

# Transmission Effects

Ideal kinematic pair (no backlash) connecting the rotation axis of the servomotor with the axis of the corresponding joint

$$c_m = I_m \dot{\omega}_m + F_m \omega_m + f r_m$$

$$f r = I \dot{\omega} + F \omega + c_l$$

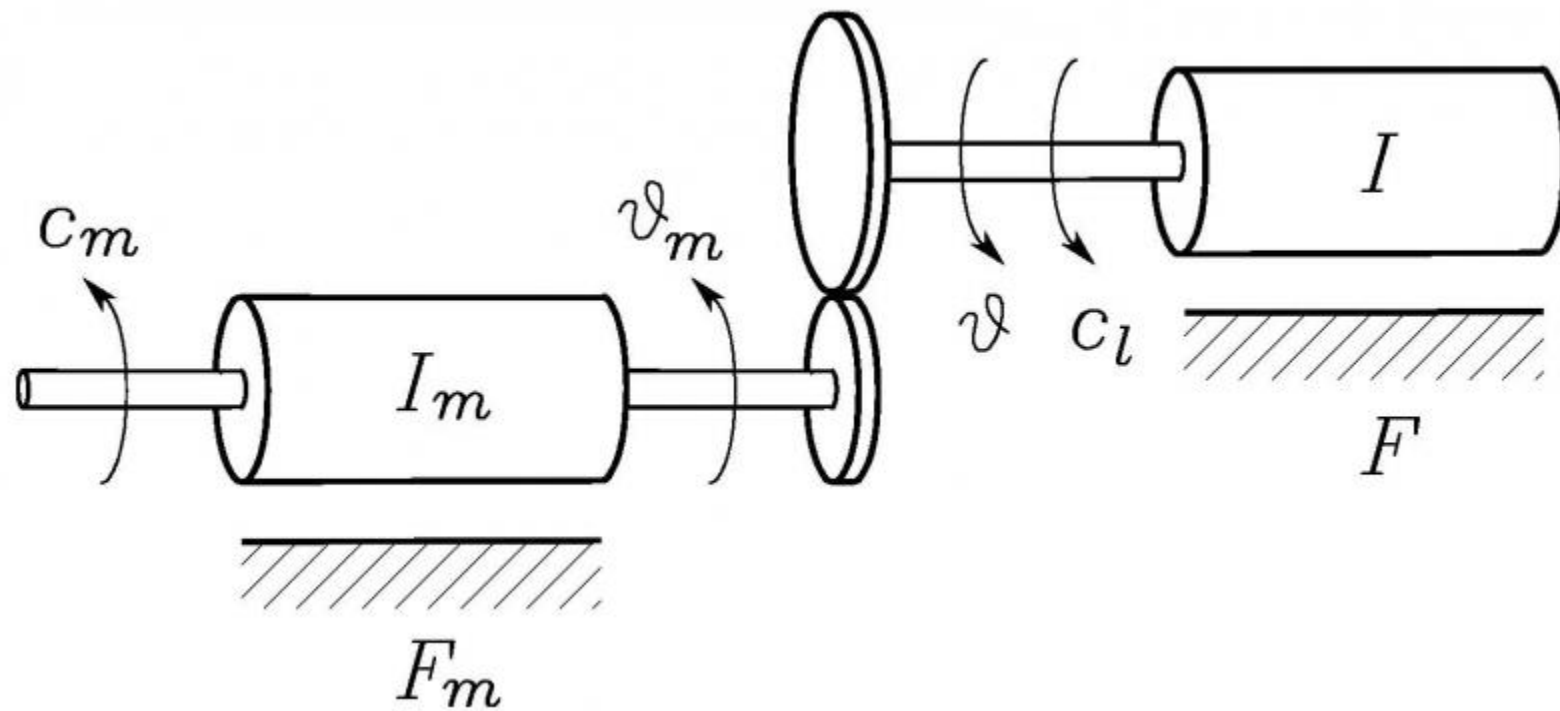
$$\Downarrow \quad k_r = \frac{r}{r_m}$$

$$c_m = I_{eq} \dot{\omega}_m + F_{eq} \omega_m + \frac{c_l}{k_r}$$

$$I_{eq} = \left( I_m + \frac{I}{k_r^2} \right) \quad F_{eq} = \left( F_m + \frac{F}{k_r^2} \right)$$

$$k_r \gg 1$$

- The inertia moment and the viscous friction coefficient of the load are reflected at the motor axis with a reduction of a factor  $1/k_r^2$
- The reaction torque is reduced by a factor  $1/k_r$



**Schematic representation of a mechanical gear**

# Example

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Pendulum actuated via mechanical gear

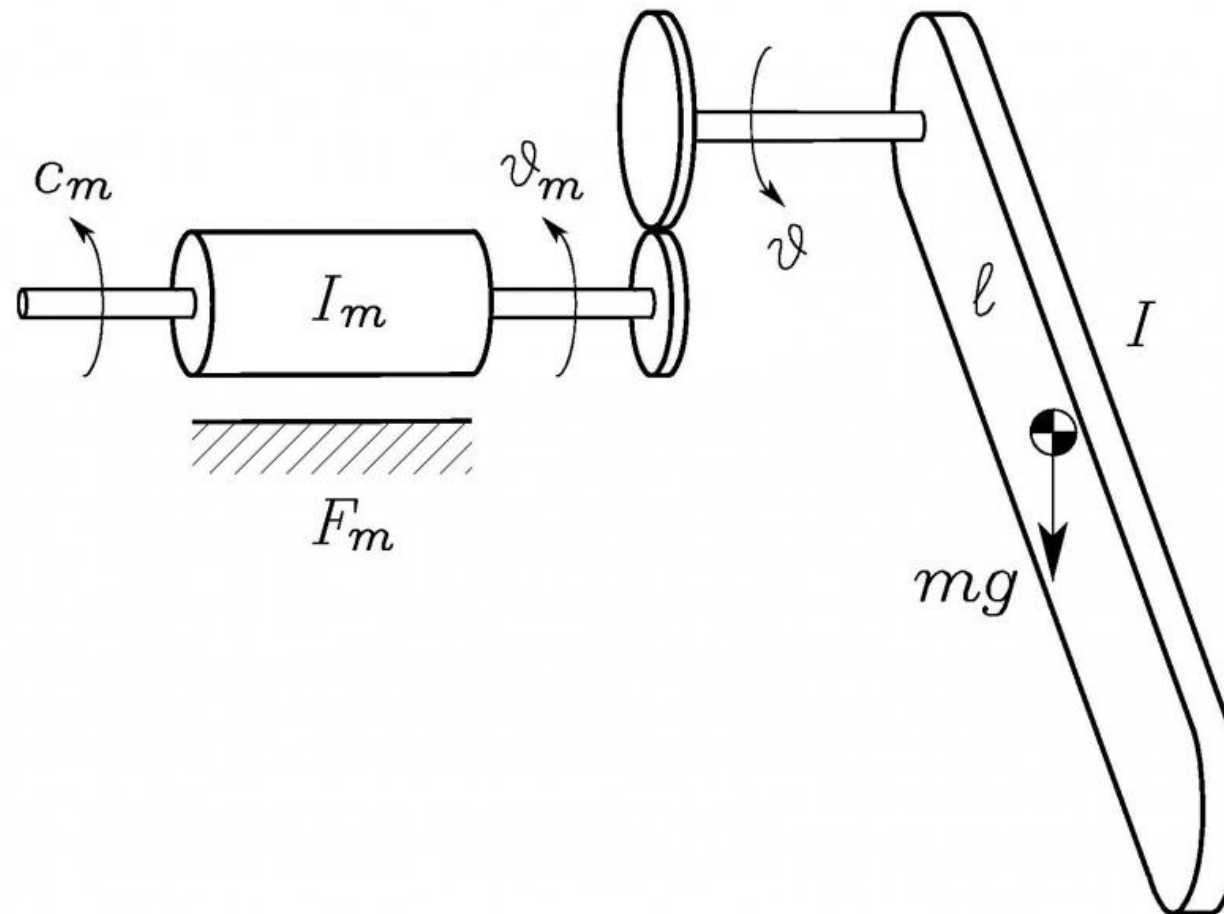
$$c_m = I_m \dot{\omega}_m + F_m \omega_m + f r_m$$

$$f r = I \dot{\omega} + F \omega + m g \ell \sin \theta$$

$\Downarrow$

$$c_m = I_{eq} \dot{\omega}_m + F_{eq} \omega_m + \left( \frac{m g \ell}{k_r} \right) \sin \left( \frac{\vartheta_m}{k_r} \right)$$

- For an  $N$ -link manipulator the nonlinear couplings between the motors of the various links will be reduced by the presence of transmissions with large reduction ratios



**Pendulum actuated via mechanical gear**

# Position Control

## SECTION 1

# Position Control

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General scheme of electric drive control

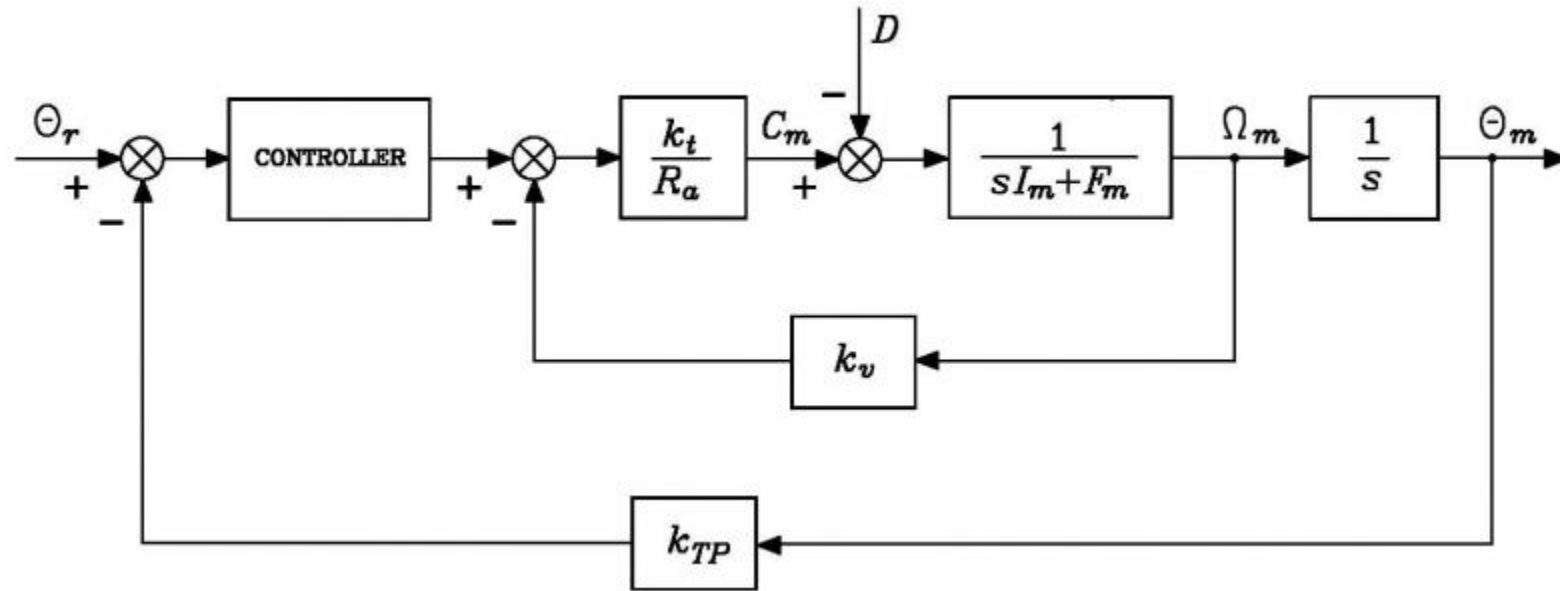
- Velocity-controlled generator

$$\frac{R_a}{k_v k_t} \ll \frac{1}{F_m}$$

Reduction of disturbance effects on the output  $\implies$  PI control

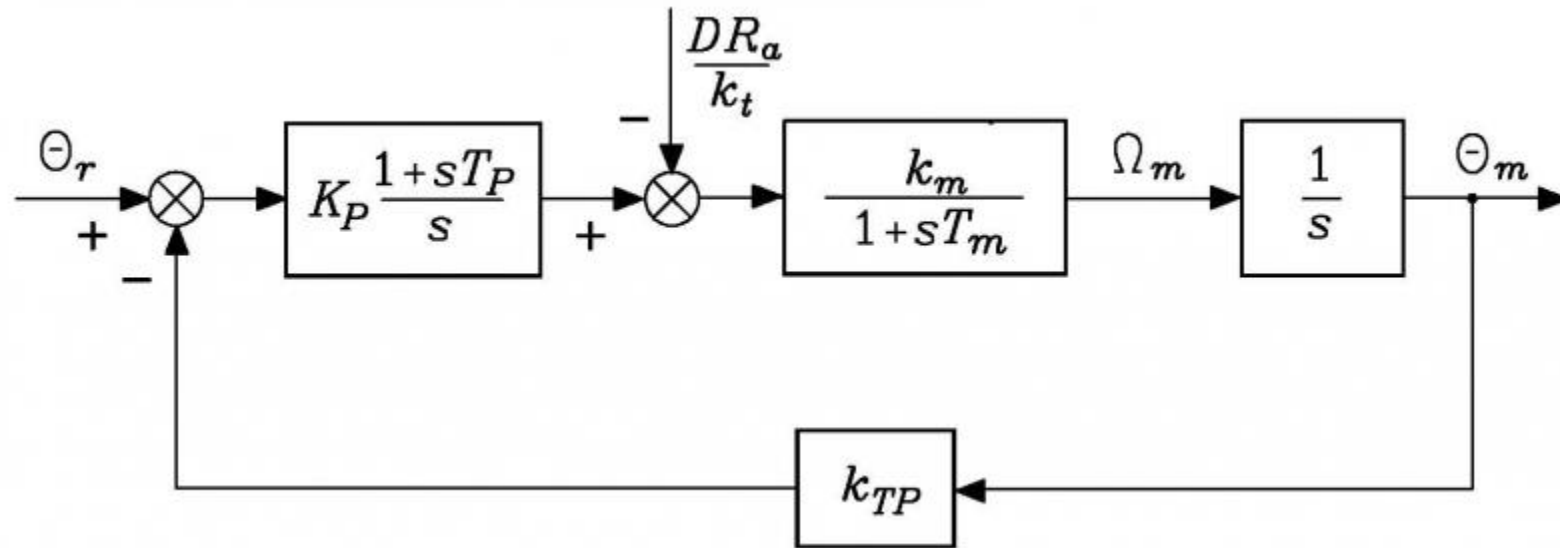
action  $k_m = \frac{1}{k_v} \quad T_m = \frac{R_a I_m}{k_v k_t}$

- $K_P$  and  $T_P$  to be keenly chosen so as to ensure stability of feedback control system and obtain a good dynamic behavior



**General block scheme of electric drive control**





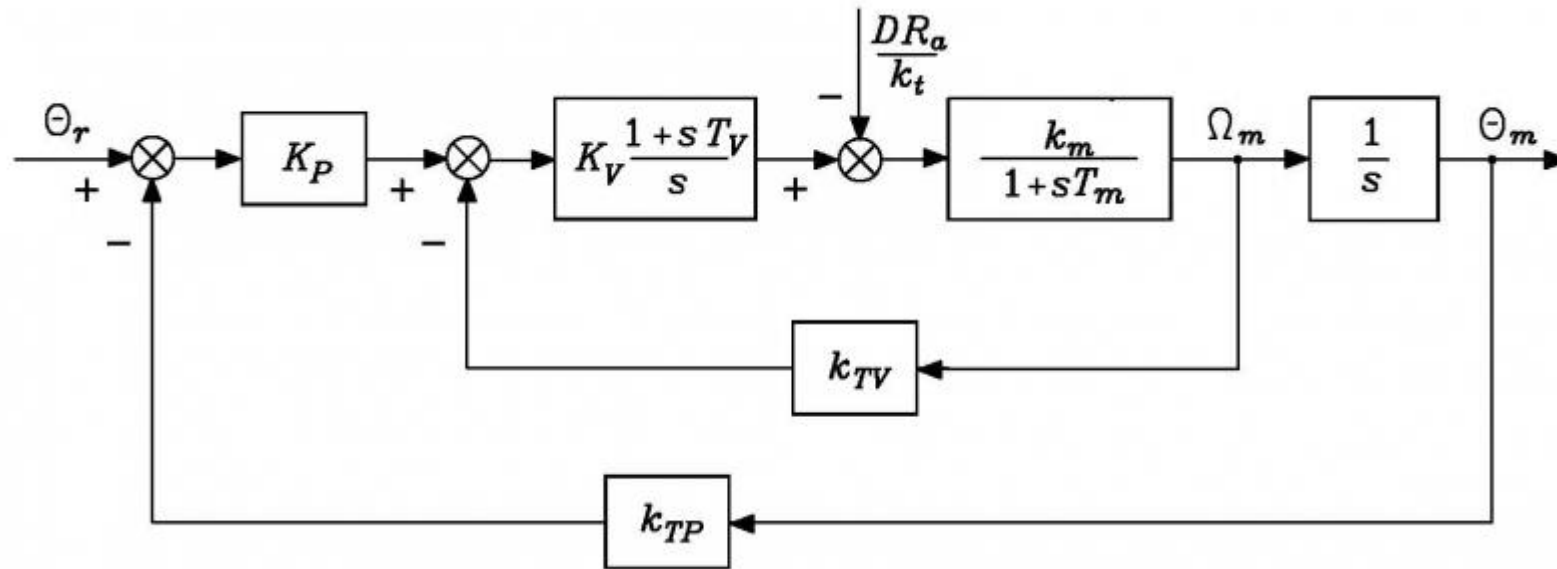
**Block scheme of position feedback control**

# Position and Velocity Feedback

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Improvement of transient response  $\Rightarrow$  Include local feedback loop based on angular velocity measurement (tachometer feedback)

- The PI control with parameters  $K_V$  and  $T_V$  is retained in the internal velocity loop so as to cancel the effects of disturbance on the position  $\vartheta_m$  at steady state
- The presence of two feedback loops is expected to lead to further reduction of disturbance during transients



**Block scheme of position and velocity feedback control**

# The Motion Control Problem

SECTION 1

# Joint space control

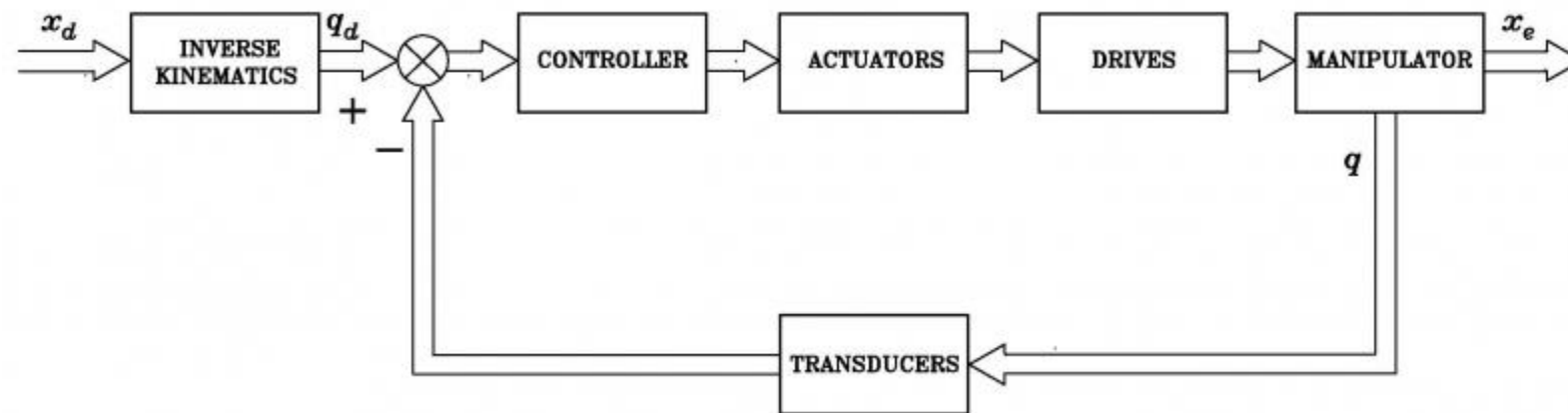
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- First stage: inverse kinematics to transform motion requirements from the operational space into the corresponding motion in the joint space
- Second stage: a joint space control scheme is designed to allow tracking of the reference motion
- Operational space variables are controlled open-loop: prone to structure uncertainty (construction tolerance, lack of calibration, gear, backlash, elasticity) or imprecision in the knowledge of the end-effector pose relative to an object to manipulate

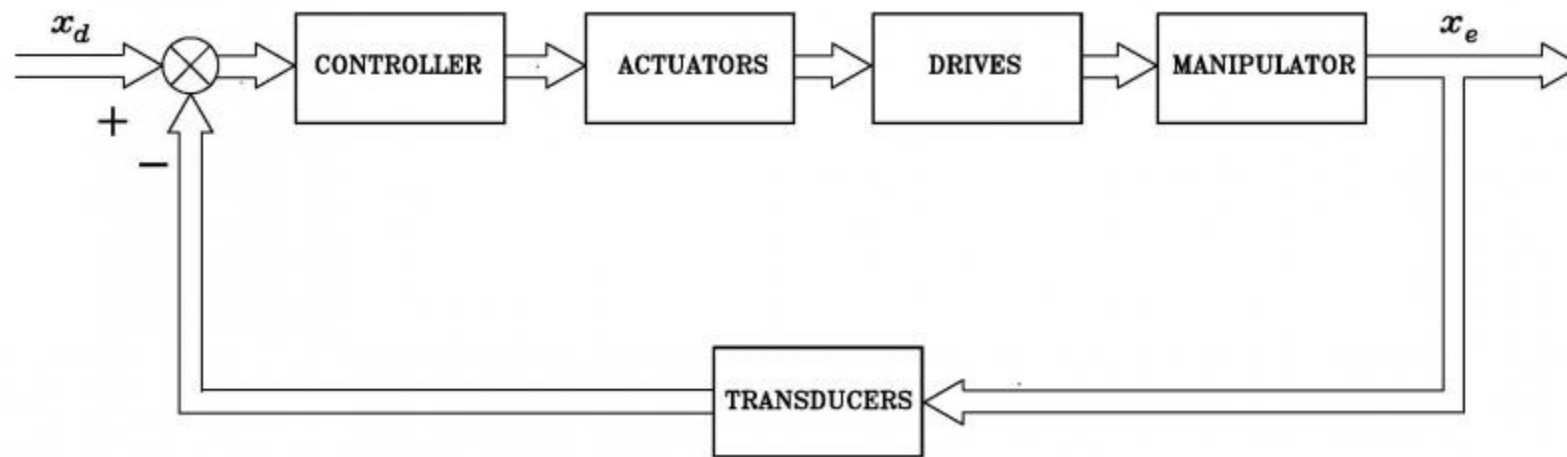
# Operational space control

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- Inverse kinematics embedded in the feedback control loop (greater algorithmic complexity)
- Direct action in the operational space
- Operational space variables typically measured through direct kinematics from measured joint space variables



**General scheme of joint space control**



**General scheme of operational space control**



# Joint Space Control

## SECTION 1

# Joint Space Control

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- Dynamic model

$$\mathbf{B}(\mathbf{q})\ddot{\mathbf{q}} + \mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}} + \mathbf{F}_v\dot{\mathbf{q}} + \mathbf{g}(\mathbf{q}) = \boldsymbol{\tau}$$

**Control**  $\equiv$  Find  $\boldsymbol{\tau}$  :  $\mathbf{q}(t) = \mathbf{q}_d(t)$

- Mechanical transmissions  $\mathbf{K}_r \mathbf{q} = \mathbf{q}_m$        $\boldsymbol{\tau}_m = \mathbf{K}_r^{-1} \boldsymbol{\tau}$

- Electric drives

$$\mathbf{K}_r^{-1} \boldsymbol{\tau} = \mathbf{K}_t \mathbf{i}_a$$

$$\mathbf{v}_a = \mathbf{R}_a \mathbf{i}_a + \mathbf{K}_v \dot{\mathbf{q}}_m$$

$$\mathbf{v}_a = \mathbf{G}_v \mathbf{v}_c$$

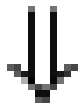
# Velocity-Controlled Manipulator

- Dynamic model of manipulator and drives
 
$$\begin{aligned} B(q)\ddot{q} + C(q, \dot{q})\dot{q} + F\dot{q} + g(q) &= u \\ F &= F_v + K_r K_t R_a^{-1} K_v K_r \\ u &= K_r K_t R_a^{-1} G_v v_c \\ K_r K_t R_a^{-1} G_v v_c &= \tau + K_r K_t R_a^{-1} K_v K_r \dot{q} \end{aligned}$$



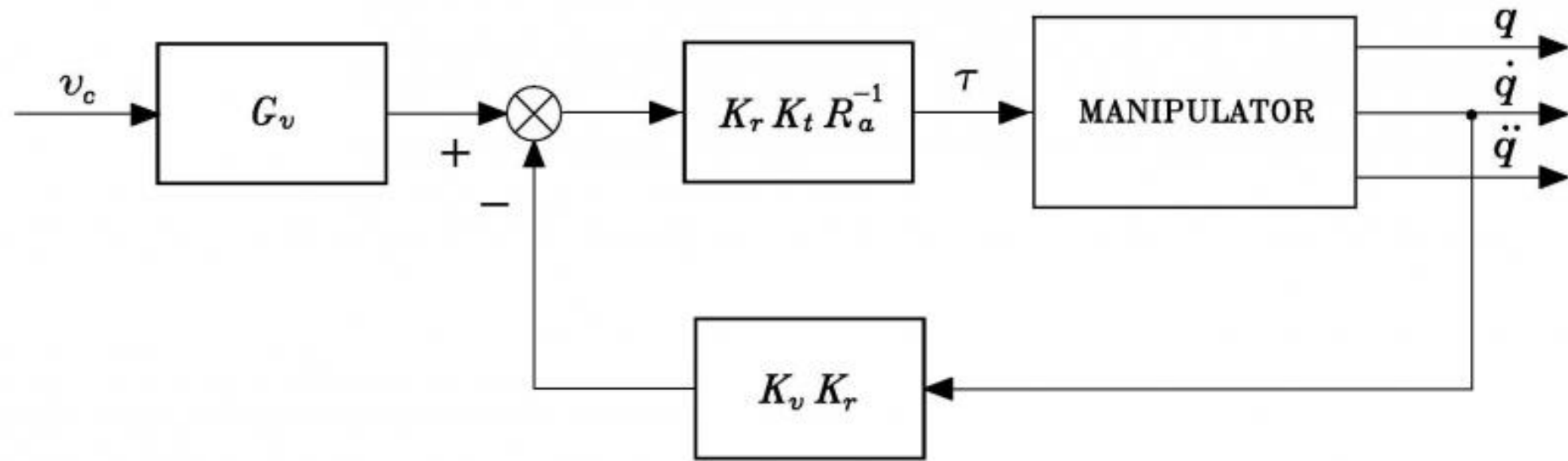
$$\tau = K_r K_t R_a^{-1} (G_v v_c - K_v K_r \dot{q}) \gg 1$$

- $K_r$  with elements
- $R_a$  with small elements (high-efficiency servomotors)
- $\tau$  not too large



- Decentralized control

$$G_v v_c \approx K_v K_r \dot{q}$$



**Block scheme of the manipulator and drives system as a voltage-controlled system**

# Torque-Controlled Manipulator

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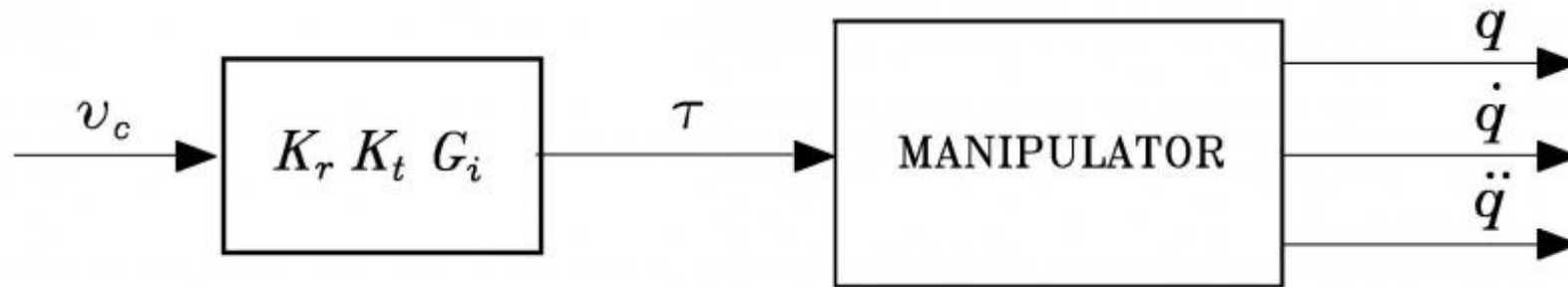
- Reduction of sensitivity to parametric variations of  $\mathbf{K}_t, \mathbf{K}_v, \mathbf{R}_a$

$$\mathbf{z}_a = \mathbf{G}_i \mathbf{v}_c$$



- Decentralized control

$$\boldsymbol{\tau} = \mathbf{u} = \mathbf{K}_\tau \mathbf{K}_t \mathbf{G}_i \mathbf{v}_c$$



**Block scheme of the manipulator and drives system as a torque-controlled system**

# Decentralized Control

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- Dynamic model at motor side

$$\mathbf{K}_r^{-1} \mathbf{B}(\mathbf{q}) \mathbf{K}_r^{-1} \ddot{\mathbf{q}}_m + \mathbf{K}_r^{-1} \mathbf{C}(\mathbf{q}, \dot{\mathbf{q}}) \mathbf{K}_r^{-1} \dot{\mathbf{q}}_m + \mathbf{K}_r^{-1} \mathbf{F}_v \mathbf{K}_r^{-1} + \mathbf{K}_r^{-1} \mathbf{g}(\mathbf{q}) = \boldsymbol{\tau}_m$$

- Average inertias

$$\begin{aligned} \mathbf{B}(\mathbf{q}) &= \bar{\mathbf{B}} + \Delta \mathbf{B}(\mathbf{q}) \\ \mathbf{K}_r^{-1} \bar{\mathbf{B}} \mathbf{K}_r^{-1} \ddot{\mathbf{q}}_m + \mathbf{F}_m \dot{\mathbf{q}}_m + \mathbf{d} &= \boldsymbol{\tau}_m \end{aligned}$$

- Viscous friction

$$\mathbf{F}_m = \mathbf{K}_r^{-1} \mathbf{F}_v \mathbf{K}_r^{-1}$$

- Disturbance

$$\mathbf{d} = \mathbf{K}_r^{-1} \Delta \mathbf{B}(\mathbf{q}) \mathbf{K}_r^{-1} \ddot{\mathbf{q}}_m + \mathbf{K}_r^{-1} \mathbf{C}(\mathbf{q}, \dot{\mathbf{q}}) \mathbf{K}_r^{-1} \dot{\mathbf{q}}_m + \mathbf{K}_r^{-1} \mathbf{g}(\mathbf{q})$$

NONLINEAR  
COUPLED

