

# MPM-Geomechanics Manual (v0.1)

MPM-Geomechanics: An open-source Material Point Method code for geomechanics.

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## 1 Slave-Master Contact

In the Slave-Master Contact method, there are two velocity fields, one for the master body  $v_{iI}^M$  and another for the slave body  $v_{iI}^S$ . The contact between two bodies occurs when the following conditions are satisfied:

- The momenta of both bodies are mapped to the same grid node  $I$ ;
- The normal velocities at the contact grid node  $I$  of the two bodies satisfy

$$(v_{iI}^M - v_{iI}^S) n_{iI}^M > 0. \quad (1)$$

The unit normal  $n_{iI}^M$  to the surface of body  $M$  at grid node  $I$  can be calculated from the mass gradient as

$$n_I = \frac{\sum_p m_p N_{Ip}}{\left| \sum_p m_p N_{Ip} \right|}. \quad (2)$$

This unit normal does not satisfy the collinearity condition  $n_{iI}^M = -n_{iI}^S$ , which leads to non-conservation of momentum and even penetration.

A collinear unit normal can be obtained by averaging the two unit normals, i.e.,

$$n_{iI}^{MS} = -n_{iI}^{SM} = \frac{n_{iI}^M - n_{iI}^S}{|n_{iI}^M - n_{iI}^S|}. \quad (3)$$

If body  $M$  is stiffer than body  $S$ , or if the surface of body  $M$  is flat/convex but the surface of body  $S$  is concave, choose the unit normal of body  $M$  as the collinear unit normal, i.e.,

$$n_{iI}^{MS} = -n_{iI}^{SM} = n_{iI}^M. \quad (4)$$

### 1.1 Contact Force

The contact force is obtained in a trial-correction approach. The momentum equation of each body is first integrated independently to obtain the trial solution as if both bodies were not in contact. If the trial solution satisfies the impenetrability condition, take the trial solution as the final true solution. If not, the contact force is applied at the contact grid nodes to prevent penetration.

The normal and tangential components of this force at the master body are defined by:

$$f_{iI}^{M,nor,k} = f_{jI}^{M,c,k} n_{jI}^{M,k} n_{iI}^{M,k}, \quad (5)$$

and

$$\min \left( \|f_{iI}^{M,tan,k}\|, \mu \|f_{iI}^{M,nor,k}\| \right) \frac{f_{iI}^{M,tan,k}}{\|f_{iI}^{M,tan,k}\|}, \quad (6)$$

respectively.

Here,

$$f_{iI}^{M,c,k} = \frac{1}{(m_I^{M,k} + m_I^{S,k}) \Delta t^k} \left( m_I^{M,k} \bar{p}_{iI}^{S,k+\frac{1}{2}} - m_I^{S,k} \bar{p}_{iI}^{M,k+\frac{1}{2}} \right), \quad (7)$$

represents the contact force for sticking contact.

## 1.2 Distance Correction

The previous contact condition may result in spurious contact. For example, when the space between two bodies approaching each other is less than twice the cell size, the previous conditions are satisfied at grid node  $I$ , identifying it as a contacted grid node, but the two bodies are not actually in contact at this time.

To avoid spurious contact, the detection condition can be improved by calculating the real distance between the two bodies.

Let  $X_I^M$  and  $X_I^S$  denote the position vectors emanating from grid node  $I$  to its closest particle in body  $M$  and body  $S$ , respectively. The distance between the two bodies can be calculated as the sum of the projections of these two position vectors onto the normal vectors of the two bodies at the grid node  $I$ :

$$D_I^{MS} = -X_I^M \cdot n_I^M - X_I^S \cdot n_I^S. \quad (8)$$

Thus, the contact detection condition can be modified as:

- The momenta of both bodies are mapped to the same grid node  $I$ ;
- $(v_{iI}^M - v_{iI}^S) n_{iI}^M > 0$ ; and
- $D_I^{MS} \leq \lambda d_c$ .

In which  $\lambda d_c$  is used to take the particle size into account,  $d_c$  is the cell size, and  $\lambda$  is set to 0.5 by default, because two particles are used initially in each direction of a cell.

## 2 Verification Problems

### 2.1 Block sliding - Slave-Master contact example

#### Introduction

This example models the sliding of a block on an inclined plane. The model was simulated six times, varying the friction coefficient  $\mu$  (0.0, 0.1, 0.2, 0.3, 0.4, and 0.5) while keeping the inclination  $\theta$  of the plane fixed at  $45^\circ$  and gravity  $g$  at  $10 \text{ m/s}^2$ .

The results obtained were compared with the analytical solution of the problem, demonstrating convergence between the results. The purpose of this model is to verify the implementation of contact, as well as to demonstrate how to build a model for this type of analysis.

#### Analytical Solution

The analytical solution for the displacement of the block in the direction of the inclined plane with slope  $\theta$  as a function of time, considering a coefficient of friction  $\mu$  and the acceleration due to gravity  $g$ , is given by:

$$d = \frac{1}{2} (g \sin \theta - \mu g \cos \theta) t^2 \quad (9)$$

#### MPM Model

The MPM model consists of two bodies, one representing the inclined plane and the other the block, both created using the keyword `cuboid`. The first has dimensions  $l_x = 32$ ,  $l_y = 14$ , and  $l_z = 2$  m, with point 1 at (0,0,0) and point 2 at (32,14,2), while the second has dimensions  $l_x = l_y = l_z = 10$  m, with point 1 at (2,2,2) and point 2 at (12,12,12).

For the elastic parameters:

$$E = 100 \times 10^6 \text{ Pa}, \quad \rho = 2500 \text{ kg/m}^3, \quad \nu = 0.3$$

The computational mesh has a cell size of  $\Delta x = \Delta y = \Delta z = 1$  m. The inclination is simulated by tilting the gravitational acceleration, which is given by  $(7.07, 0.0, -7.07) \text{ m/s}^2$ . The planes  $X_0$ ,  $Y_0$ ,  $Z_0$ ,  $X_n$ ,  $Y_n$ , and  $Z_n$  are defined as sliding, i.e., only tangential displacements are allowed.

#### Input File

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{
  "time": 2.0,
  "critical_time_step_multiplier": 0.15,
  "gravity": [7.07, 0.0, -7.07],
  "results":
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    "print": 25,
    "material_point_results": [ "id", "displacement", "material", "active", "pressure", "e"
  ],
  "mesh":
  {
    "cells_dimension": [ 1, 1, 1 ],
    "cells_number": [ 32, 14, 14 ],
    "origin": [ 0, 0, 0 ],
    "boundary_conditions":
    {
      "plane_X0": "sliding",
```

```

        "plane_Y0": "sliding",
        "plane_Z0": "sliding",
        "plane_Xn": "sliding",
        "plane_Yn": "sliding",
        "plane_Zn": "sliding"
    }
},
"material":
[
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        "type": "elastic",
        "id": 1,
        "young": 100e6,
        "density": 2500,
        "poisson": 0.3
    },
    {
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        "id": 2,
        "young": 100e6,
        "density": 2500,
        "poisson": 0.3
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],
"body":
[
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        "type": "cuboid",
        "id": 1,
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        "point_p2": [ 32, 14, 2 ],
        "material_id": 1
    },
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        "id": 2,
        "point_p1": [ 2, 2, 2 ],
        "point_p2": [ 12, 12, 12 ],
        "material_id": 2
    }
],
"contact_manager": {
    "active": true,
    "real_distance_correction_coefficient": 0.5
},

"contact": [
    {
        "id": 1,
        "friction": 0.5,
        "master_id": 1,

```

```

        "slave_id": 2,
        "normal_type": "master"
    }
]
}

```

## MPM Result Comparison

The MPM numerical results show good agreement with the analytical solution, with small deviations for the smaller values of friction coefficient.

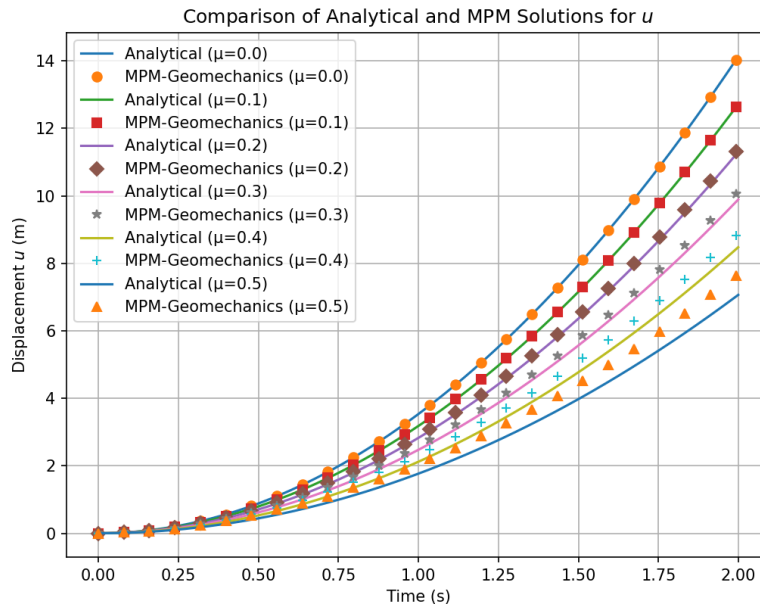


Figure 1: Verification of displacements obtained with MPM simulation.