

Magnetic Levitation Video & Requirements

http://www.youtube.com/watch?v=GVHwvMadvTg

- Academic System Requirements
 - Learn About PID Control
 - Learn About Electronics / Digital Control
 - Learn About Linearization
- Industrial System Requirements
 - Performance ~ "Small" Errors in Position
 - Disturbance Rejection ~ Zero Steady Error & "Fast" Response
 - Noise Suppression
 - Robustness
 - Performance = Errors Don't Depend Strongly on Mass Being Levitated
 - Stability = Closed-Loop Stability Not Compromised by System Variations
- We Will Design to Usual Nominal Targets
 - Natural Frequency / Damping Ratio / Zero Steady Error



ESE 505 & MEAM 513: Magnetic Levitation Control

High-Level Block Diagram

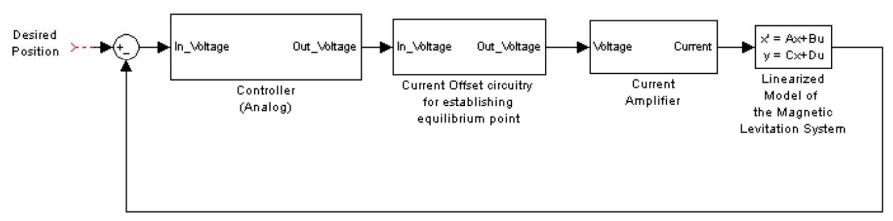


Figure 2: Block diagram of our system



Analog Implementation!

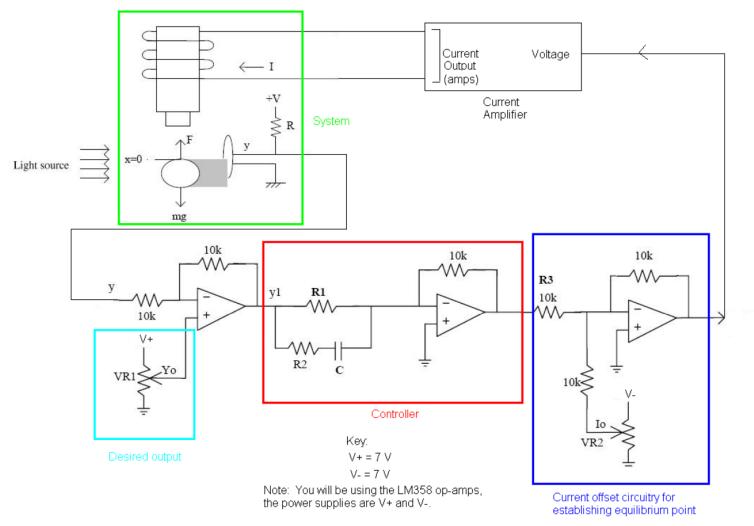
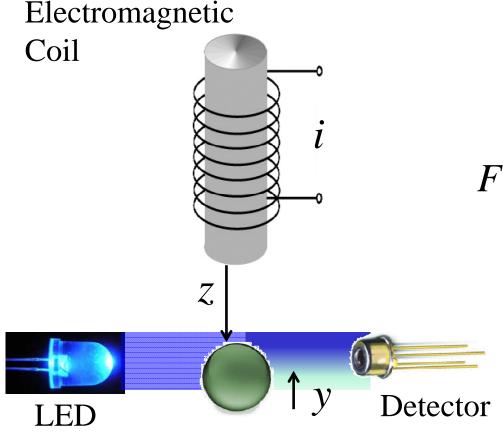


Figure 3: You will be picking values for R1, R2 and C for stable equilibrium



Equations of Motion



$$m\frac{d^2z}{dt^2} = mg - F(z,i)$$

$$F(z,i) = \frac{\mu_o(Ni)^2 A}{4z^2} = k \frac{i^2}{z^2}$$

N = # Turns

 μ_o = Permeability of Air

A =Pole Face Area

$$y = z_o - z$$

Force Model Adapted From http://neil-mclagan.net/Maglev/NSM-CEVS-Chap-2.pdf



Model in State Space Form & Linearization

$$x_{1} = z
x_{2} = \frac{dz}{dt}
u = i
y = x_{1o} - x$$

$$\frac{dx_{1}}{dt} = x_{2}
\frac{dx_{1}}{dt} = x_{2}
\frac{dx_{2}}{dt} = y - \frac{1}{m}F(x_{1}, u) = y - \frac{ku^{2}}{mx_{1}^{2}}
u_{0} = \frac{mgx_{1o}^{2}}{k}$$

LINEARIZATION

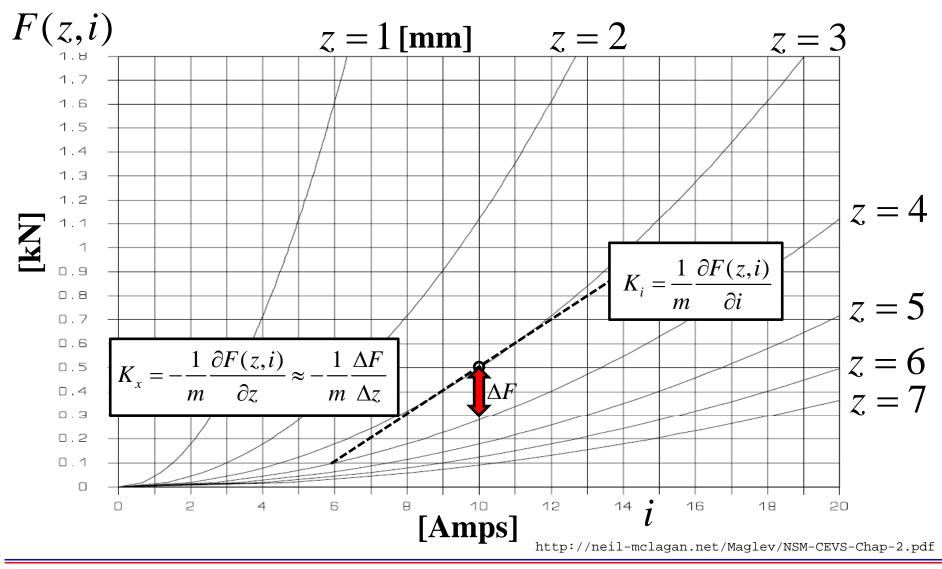
$$\begin{bmatrix} A \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -\frac{1}{m} \frac{\partial F}{\partial x_1} & 0 \end{bmatrix}_o = \begin{bmatrix} 0 & 1 \\ \frac{2ku_o^2}{mx_{1o}^3} & 0 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ K_x & 0 \end{bmatrix} \quad \begin{bmatrix} B \end{bmatrix} = \begin{bmatrix} 0 \\ -\frac{1}{m} \frac{\partial F}{\partial u} \end{bmatrix}_o = \begin{bmatrix} 0 \\ -\frac{2ku_o}{mx_{1o}^2} \end{bmatrix} = \begin{bmatrix} 0 \\ -K_i \end{bmatrix}$$

$$\begin{bmatrix} C \end{bmatrix} = \begin{bmatrix} -1 & 0 \end{bmatrix} \qquad \begin{bmatrix} D \end{bmatrix} = \begin{bmatrix} 0 \end{bmatrix} \qquad K_x = -\frac{1}{m} \frac{\partial F(z, i)}{\partial z} = \frac{2ki_o^2}{mz_o^3} \qquad K_i = \frac{1}{m} \frac{\partial F(z, i)}{\partial i} = \frac{2ki_o}{mz_o^2}$$

$$\frac{Y(s)}{U(s)} = G_P(s) = C(sI - A)^{-1}B + D = \frac{K_i}{s^2 - K_x}$$

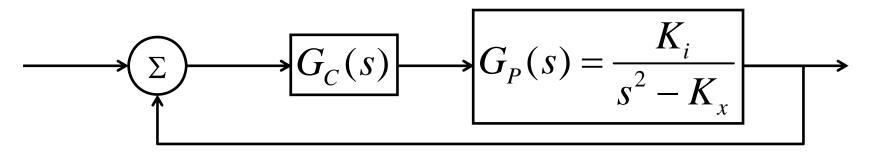


Magnetic Force (For Large-Scale Experiment)





Block Diagram & Compensators



Start with Proportional Control

$$G_C(s) = K$$

Lead Shaping = (P+D) Through Low-Pass Filter

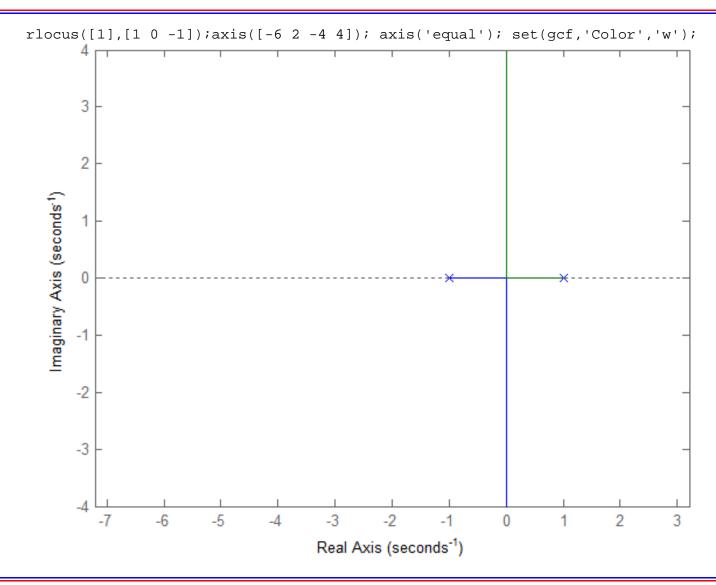
$$G_C(s) = K \frac{\tau_2 s + 1}{\tau_1 s + 1} \quad \tau_2 > \tau_1$$

Lead-Lag = Generalization of P+I+D

$$G_C(s) = K \frac{\tau_2 s + 1}{\tau_1 s + 1} \frac{s + z}{s + p}$$
 $\tau_2 > \tau_1$ $z > p$



Typical Root Locus for Proportional Feedback

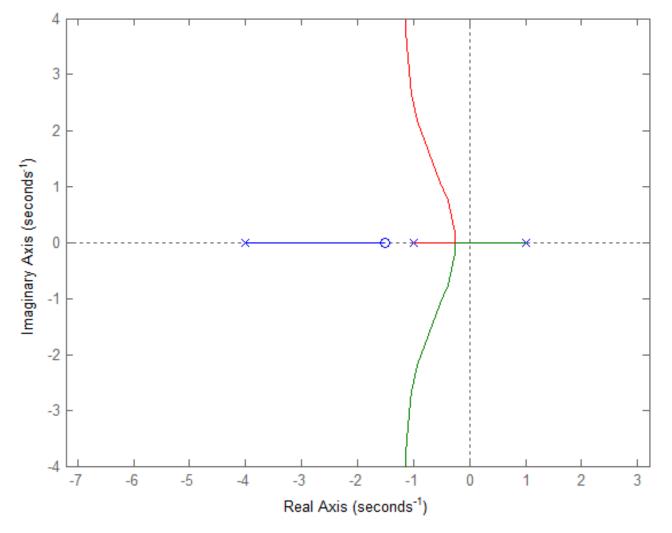




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Typical Root Locus for Lead Compensation

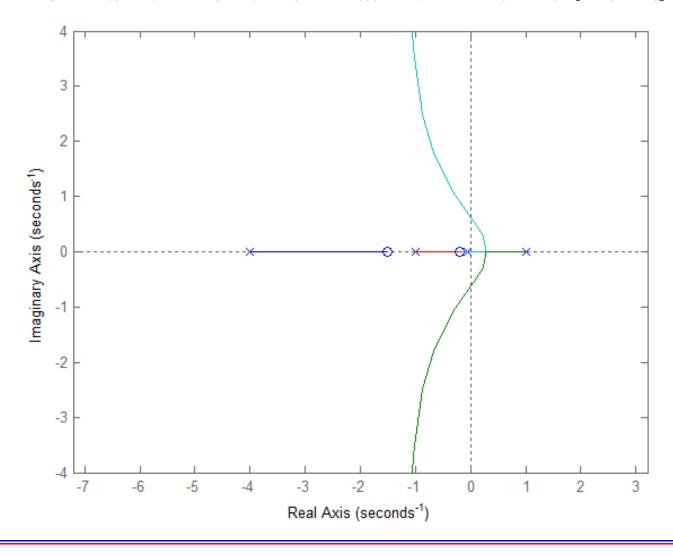
rlocus([1 1.5],conv([1 0 -1],[1 4]));axis([-6 2 -4 4]); axis('equal'); set(gcf,'Color','w');





Typical Root Locus for Lead-Lag Compensation

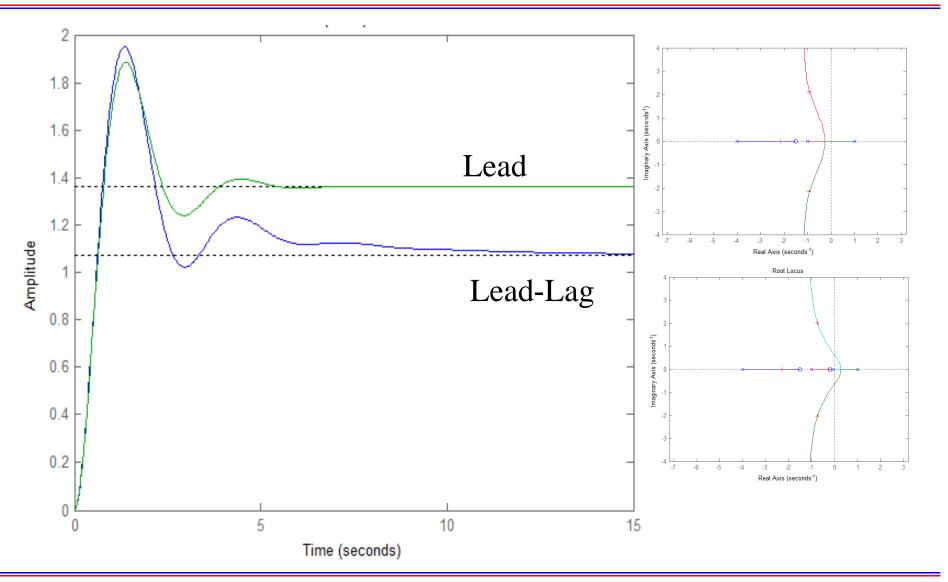
rlocus(conv([1 1.5],[1 0.2]),conv([1 0 -1],conv([1 4],[1 0.05])));axis([-6 2 -4 4]); axis('equal'); set(qcf,'Color','w');





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Closed-Loop Time Responses (K=10)





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