

PERSONAL NOTES

NONLINEAR F.E.A.

Hanfeng Zhai

Disclaimer: These notes are intended solely for personal reference and study purposes. They represent my own understanding of the course material and may contain errors or inaccuracies. The content presented here should not be considered as an authoritative source, and reliance solely on these materials is not recommended. If you notice any materials that potentially infringe upon the copyright of others, please contact me at hz253@cornell.edu so that appropriate action can be taken. Your feedback is greatly appreciated.

#FEM (Nonlinear)

▷ What is FEA?

Method to solve PDE.

▷ Why Nonlinear FEA important?

large deformation.

(linearize the strain).

↓
plastic deformation.

↳ material response
hyperelastic, etc. . . .

▷ tools for linear FEA. - relevant?

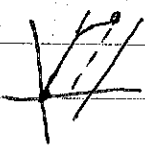
Strong form → weak form

↓
discretize

↓
Shape function

* ~~At~~ local support.

Key difference:
the solution



Prescribe the
behavior within the
element.

↓
(interpolation)

↳ derivatives

↓
Solution → assembly.
↳ postprocessing.

← Gauss Quadrature
(integration).

Sources of nonlinearities in solids.

• geometric nonlinearities.

• deformation

• material responses

• instabilities

• BCs.

• coupled problem.

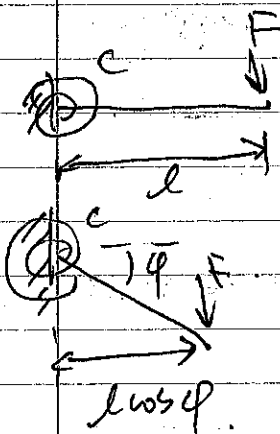
↖ force, multiphysics

- large deformation (displ.) of rigid beam

⇒ Example:

~ rigid beam, rotational spring stiffness c .

$$\sum M = 0 \rightarrow Fl \cos \varphi = c \varphi$$

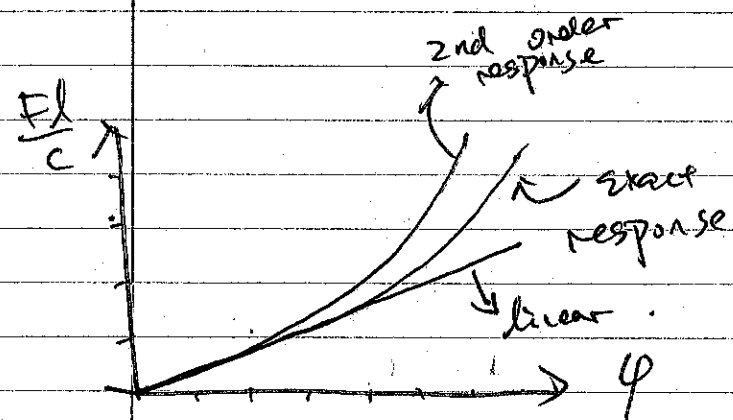


For small rotations $\cos \varphi \rightarrow 1$:

$$F = c \varphi / l \quad \varphi \rightarrow 0$$

to capture the nonlinearity, → 2nd order theory
expand the function into Taylor series:

$$\cos \varphi \approx 1 - \frac{\varphi^2}{2}$$



$$F = \frac{c \hat{\varphi}}{2(1 - \frac{\hat{\varphi}^2}{2})}$$

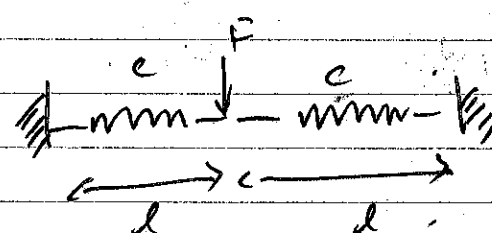
"2nd order"

* geometric nonlinearities.

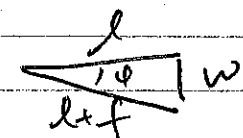
general motion.

* this is not deformation - no strain tensor involved.

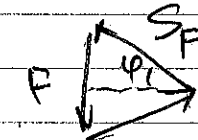
large deformation of elastic system.



$$\sum \vec{F} = 0$$



↪ elongation.



- kinematics

- equilibrium

- constitutive law

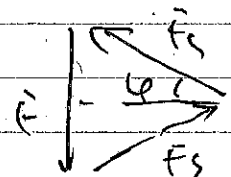
kinematics

$$w^2 + l^2 = (1+f)^2$$

$$\sin \varphi = \frac{w}{1+f}$$

equilibrium

$$2F_s \sin \varphi = F$$



constitutive law:

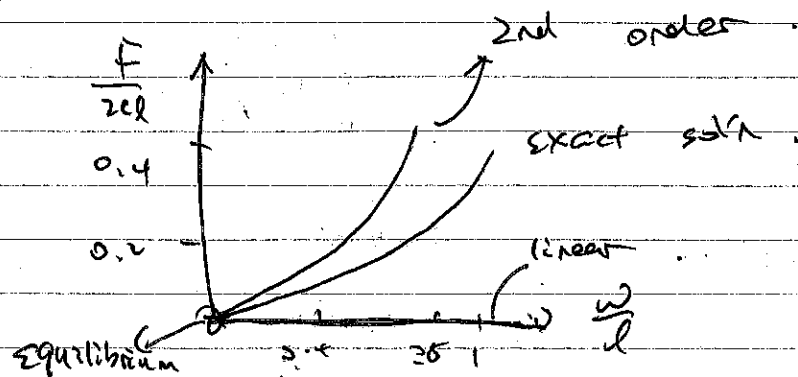
$$F_s = c f \leftarrow \text{elongation}$$

↑
spring constant

exact sol'n:

$$\frac{w}{l} \left[1 - \frac{1}{\sqrt{1 + \left(\frac{w}{l}\right)^2}} \right] = \frac{F}{2cl}$$

the exact sol'n:



Taylor expansion:

$$\frac{1}{\sqrt{1 + \left(\frac{w}{l}\right)^2}} \approx 1 - \frac{1}{2} \left(\frac{w}{l}\right)^2$$

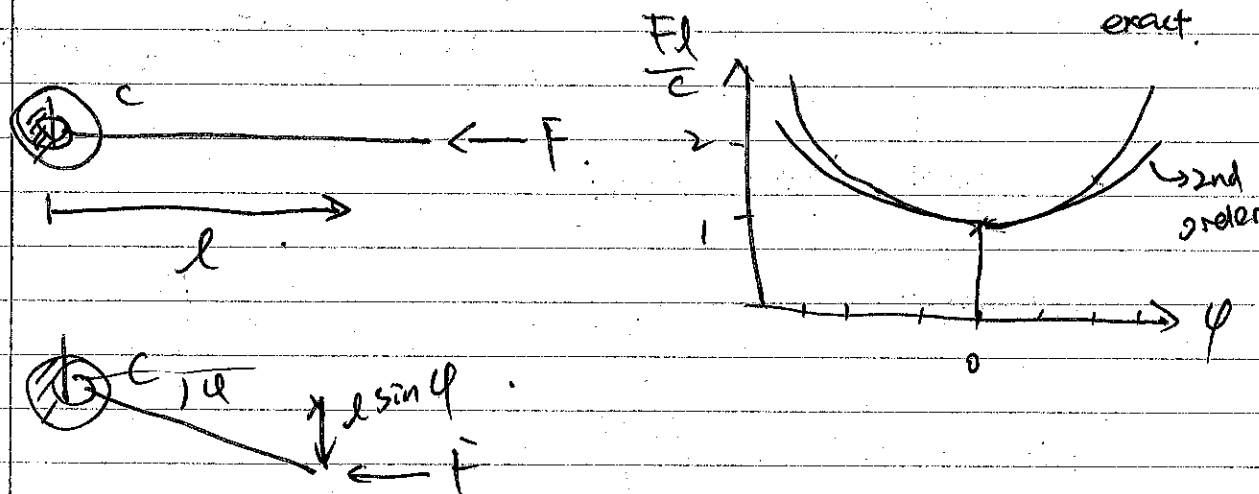
small values of $w \rightarrow \frac{w}{l} \ll 1$

$$\frac{w}{l} \left[\frac{1}{2} \left(\frac{w}{l}\right)^2 \right] = \frac{F}{2cl}$$

source of nonlinearities: deformation & rotation

Bifurcation

$$Fl \sin \varphi = c \varphi$$



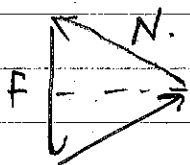
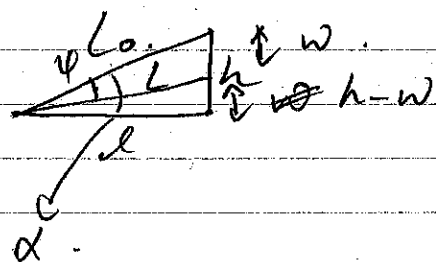
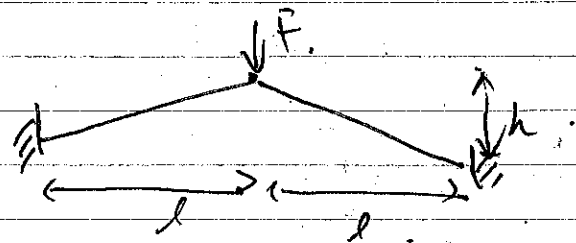
Approximation:

$$\text{Taylor expansion: } \sin \varphi \approx \varphi - \frac{\varphi^3}{6}$$

$$\& \quad \frac{1}{1-x} \approx 1+x$$

Snap-through.

Geometry: $(h-w)^2 + l^2 = L^2$ & $L^2 + l^2 = L_0^2$



from this we can write length change:

$$f = L - L_0 = l \left[\sqrt{1 + \left(\frac{h-w}{l}\right)^2} - \sqrt{1 + \left(\frac{h}{l}\right)^2} \right]$$

↓ final length
 ↑ initial

Equilibrium: $N \sin(\alpha - \varphi) = -\frac{F}{2}$

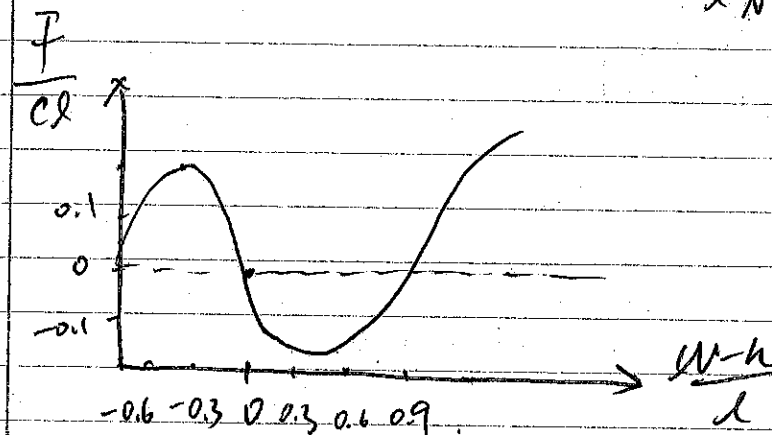
$$\sin(\alpha - \varphi) = \frac{h-w}{L}$$

$$\hookrightarrow N \frac{h-w}{L} = -\frac{F}{2}$$

Constitutive law: $F_s = cf \rightarrow$ final expression

$$c(h-w) \frac{L-L_0}{L} = -\frac{F}{2} \rightarrow$$

$$\frac{wh}{l} \left[1 - \frac{L_0}{l \sqrt{1 + \left(\frac{h-w}{l}\right)^2}} \right] - \frac{F}{2cl} = 0$$



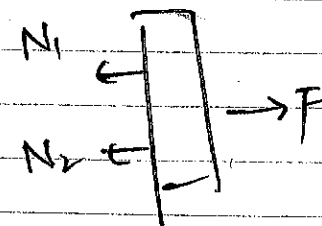
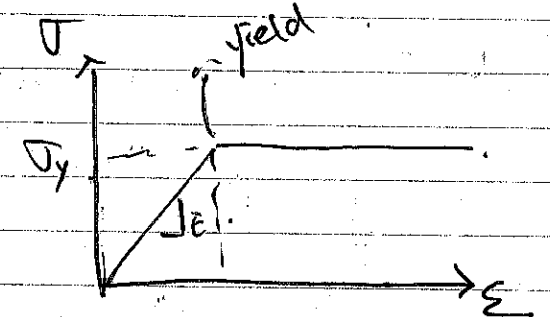
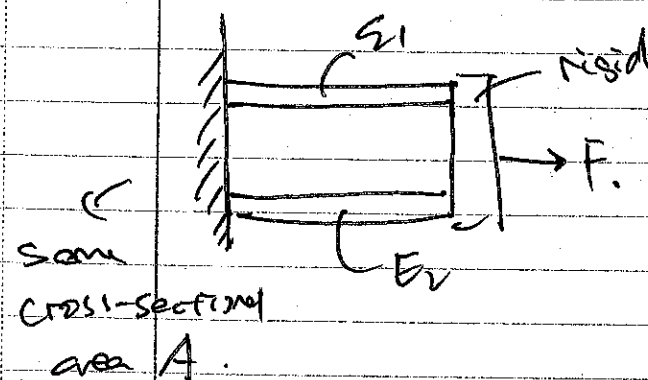
(*) F = applied force

Material nonlinearities

elastic moduli: $E_1 = 2E_2 = 2E$

their yield stresses:

$$\sigma_{y1} = 3\sigma_{y2} = 3\sigma_y$$



$$N_1 + N_2 = F$$

$$\sigma_1 + \sigma_2 = \frac{F}{A}$$

$$u_1 = u_2 = u, \quad \epsilon = \frac{u}{l}$$

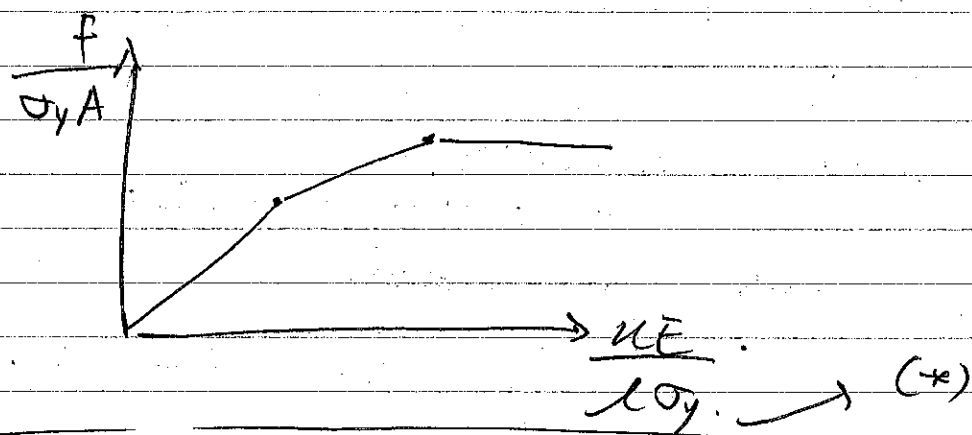
connect thru Hookes law.

$$\sigma_i = E \epsilon = E \frac{u}{l}$$

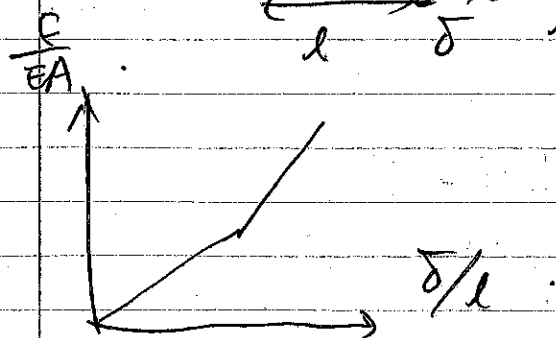
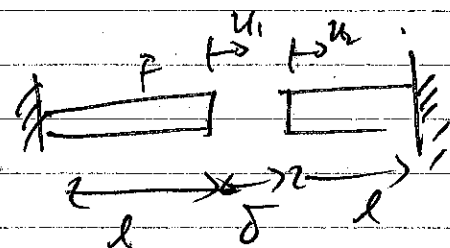
$$u = \frac{Fl}{(E_1 + E_2)A}$$

$$\sigma_y < \sigma_{y1}$$

Bar 2 yields $\rightarrow F = 3A\sigma_y$



BC induced nonlinearity



HW: notes

$$\frac{\partial (A_{ik}^{-1} A_{kj})}{\partial A_m} = -A_{ik}^{-1} \frac{\partial A_{ik}^{-1}}{\partial A_m} A_{kj} + A_{ik}^{-1} \frac{\partial A_{kj}}{\partial A_m}$$

$$\rightarrow \frac{\partial A^{-1}}{\partial A} \quad \text{hint: } A_{ik}^{-1} A_{kj} = \delta_{ij}$$

$$\frac{\partial A^{-1} A}{\partial A_{kj}} = \frac{\partial A_{ik}^{-1}}{\partial A_{kj}} A_{ij} + A_{ik}^{-1} \frac{\partial A_{ij}}{\partial A_{kj}} = \delta_{ij}$$

$$\frac{\partial A_{ij}^{-1}}{\partial A_{pq}} = -A_{ip}^{-1} A_{jq}^{-1} \quad \frac{\partial (ab)}{\partial c} = \frac{\partial a}{\partial c} b + \frac{\partial b}{\partial c} a$$

$$\frac{\partial \det A}{\partial A} = \det(A) \cdot A^{-T}$$

$$\frac{\partial \det(A_{ij})}{\partial A_{kl}} =$$

Prob 1. $\frac{\partial (A^{-1} A_K)}{\partial A_K}$

$$= \frac{\partial A_{ik}^{-1}}{\partial A_{pq}} \cdot A_{kj} + \frac{\partial A_{kj}}{\partial A_{pq}} \cdot A_{ik}^{-1}$$

$$\frac{\partial \delta_{ij}}{\partial A_{pq}} = \partial A_{kj} \quad \text{or} \quad \frac{\partial \delta_{ij}}{\partial A_{pq}} = \frac{\partial \delta_{ij}}{\partial A_{pq}}$$

$$A_{ik} A_{kj}^{-1} = \delta_{ij}$$

$$\frac{\partial A_{ij}^{-1}}{\partial A_{pq}} A_{jk} + A_{ij}^{-1} \frac{\partial A_{jk}}{\partial A_{pq}} = \frac{\partial A_{ik}^{-1}}{\partial A_{pq}} A_{kk} + A_{ik}^{-1} \frac{\partial A_{kk}}{\partial A_{pq}}$$

$$\begin{aligned} \frac{\partial A_{ij}^{-1}}{\partial A_{pq}} &= -A_{il}^{-1} \frac{\partial A_{lk}}{\partial A_{pq}} A_{kj}^{-1} \\ &= -A_{il}^{-1} \delta_{lp} \delta_{kq} A_{kj}^{-1} \\ &= -A_{ip}^{-1} A_{qj}^{-1} \end{aligned}$$

Prob. 2

$$\frac{\partial \det(\underline{A})}{\partial \underline{A}} = \frac{\partial \det(\underline{A})}{\partial A_{ij}} = \frac{\partial \|\underline{A}\|}{\partial A_{ij}}$$

$$= \sum_i \text{sgn} \Pi a$$

direct notation.

$$\det A = \begin{vmatrix} a_{11} & & \\ & \dots & \\ & & \end{vmatrix}$$

$$\det \underline{A} = \|A_{ij}\|$$

$$\sum_i \sum_l \frac{\partial A_i}{\partial A_l} \cdot \frac{\partial A_{kl}}{\partial A_{ij}}$$

$$\frac{\partial \det A}{\partial A_{ij}} = \sum_k^n \sum_l^n C_{kl} \delta_{ik} \delta_{jl} = L_3 A^T$$

$$\frac{\partial \det A}{\partial A_{ij}} = \sum_k \sum_l$$

$$f(\underline{A}) = \det(\underline{A}).$$

27

1 variant:

$$\left(\frac{\partial \mathcal{L}}{\partial A} \right) = \underline{\underline{I}}$$

$$\left\{ \frac{2L_2}{5A} = L_1 L_2 - A^2 \right.$$

$$L \frac{\partial L}{\partial A} = (A^T)^L - L A^T$$

$$+ \underline{\underline{L_2 T}}$$

$$= \underline{\underline{L_3 A^T}}$$

$$\det(A)$$

34

$$A_1 A_2 - A_2 A_1$$

$$\det(A) = \frac{1}{6} \epsilon_{ijk} \epsilon_{abc} A_{ia} A_{jb} A_{kc}$$

$$\frac{\partial}{\partial A_{ij}} \sum_k \epsilon_{ijk} \epsilon_{abc} A_{ia} A_{jb} A_{kc}$$

$$\frac{\partial}{\partial A_{ij}}$$

$$\frac{\partial \det A}{\partial A_{ij}} = (A_{ji}^T)^2 - [A_{ji}^T + I_r]$$

$$= (A_{ji}^T)^2 - \text{tr } A_{ji} A_{ji}^T$$

$$+ \frac{1}{2} [(\text{tr } A_{ji})^2 - \text{tr } A_{ji}^2]$$

$$L_3 = \det A_{ji}$$

$$L_1 = \text{tr } A_{ji} = A_{ji}^T - \text{tr } A_{ji} A_{ji}^T$$

$$+ \frac{1}{2} [\sim]$$

$$L_2 = \frac{1}{2} [(\text{tr } A)^2 - \text{tr } (A^2)]$$

$$\downarrow$$

$$= \det A_{ji} A_{ji}^T$$

$$= A_{ji}^T - \text{tr } A_{ji} A_{ji}^T + \frac{1}{2} [(\text{tr } A_{ji})^2 - \text{tr } A_{ji}^2]$$

$$\frac{\partial (A_{11} A_{22} - A_{12} A_{21})}{\partial} = A_{22} - A_{11}$$

Prob. 3

$$\underline{D} = \underline{a} \otimes \underline{b}$$

$$= \begin{bmatrix} a_1 b_1 & a_1 b_2 & a_1 b_3 \\ a_2 b_1 & a_2 b_2 & a_2 b_3 \\ \vdots & \vdots & \vdots \end{bmatrix}$$

$$\frac{\partial \det(A)}{\partial A} = \frac{\partial \det(A)}{\partial A} \frac{\partial \det(A)}{\partial A}$$

$$\frac{\partial \det(A)}{\partial A} = \lim_{h \rightarrow 0} \frac{\det(A+h) - \det A}{h}$$

$$= \det A \lim_{h \rightarrow 0} \frac{\det[A^{-1}(A+h)] - 1}{h}$$

$$= \det$$

$$\det(A) = \det A$$

Prob 4. $\underline{A} = \alpha(\underline{I} - \underline{e}_1 \otimes \underline{e}_1) + \beta(\underline{e}_1 \otimes \underline{e}_2 + \underline{e}_2 \otimes \underline{e}_1)$

compute - eigen values

- eigenvectors

we can first write out the matrix form

$$\begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} = \begin{bmatrix} \alpha & \beta \\ \beta & \alpha \end{bmatrix}$$

Prob 5. - vector form

$$\oint_C \underline{\nabla} \times \underline{F} d\underline{s} = \iint_S \text{curl } \underline{F} d\underline{S}$$

$$\iint_S \text{curl } \underline{F} d\underline{S} = \iiint_V \text{div}(\text{curl } \underline{F}) dV$$

$$\therefore \text{curl}(\text{curl } \underline{F}) = \nabla(\nabla \cdot \underline{F}) - \nabla^2 \underline{F}$$

$$\iiint_V \text{div}(\text{curl } \underline{F}) dV = \iiint_V [\nabla(\nabla \cdot \underline{F}) - \nabla^2 \underline{F}] dV$$

$$\iiint_V [\nabla(\nabla \cdot \underline{F}) - \nabla^2 \underline{F}] dV = \iint_S (\nabla \cdot \underline{F}) \underline{n} dS - \iint_S \underline{F} \cdot \underline{n} dS$$

$$\oint_C \underline{\nabla} \times \underline{F} d\underline{s} = \iint_S (\nabla \cdot \underline{F}) \underline{n} dS - \iint_S \underline{F} \cdot \underline{n} dS$$

Prob. 5

$$\oint_C \phi d\underline{x} = \int_S \underline{n} \times \text{grad } \phi dS$$

$$\oint_C \underline{u} \times d\underline{x} = \int_S [(\text{div } \underline{u}) \underline{n} - (\text{grad } \underline{u}) \underline{n}] dS$$

$$\oint_C \phi d\underline{x} = \int_S \underline{n} \times \text{grad } \phi dS$$

$$\oint_C \underline{u} \times d\underline{x}$$

$$\int_C \underline{A} \cdot d\underline{x} = \int_S \underline{A} \times d\underline{S}$$

$$\oint_C \phi d\underline{x} = \int_S \underline{n} \times \text{grad } \phi dS$$

Proof for 1:

$$\begin{aligned} \oint_C \phi d\underline{x} &= \int_C d(\underline{x} \cdot \text{grad } \phi) \\ &= \int_C d\underline{x} \times \text{grad } \phi dS \end{aligned}$$

$$= \int_S \underline{n} \times \text{grad } \phi dS$$

$d(\underline{x} \cdot \text{grad } \phi)$
 $d\underline{x} \cdot \text{grad } \phi$
 scalar + \underline{x}

Proof for 2:

$$\int_C \underline{u} \times d\underline{x} = \int_C d(\underline{x} \times \underline{u})$$

$$= \int_S d\underline{x} \times \underline{u} dS$$

$$= \int_S [\operatorname{div} \underline{u} \underline{n} - \operatorname{grad} \underline{u} \underline{n}] dS$$

Gauss's Theorem

$$\int_S \underline{\phi} \cdot d\underline{S} = \int_V \operatorname{div} \underline{\phi} dV$$

$$\int_C \underline{\phi} d\underline{x} = \int_S \underline{n} \times \operatorname{grad} \underline{\phi} dS$$

Consider efflux $dE: dE = \underline{\phi} \cdot d\underline{S}$

$$\operatorname{div} \underline{\phi} dV = dE = \underline{\phi} \cdot d\underline{S}$$

Stokes' Theorem

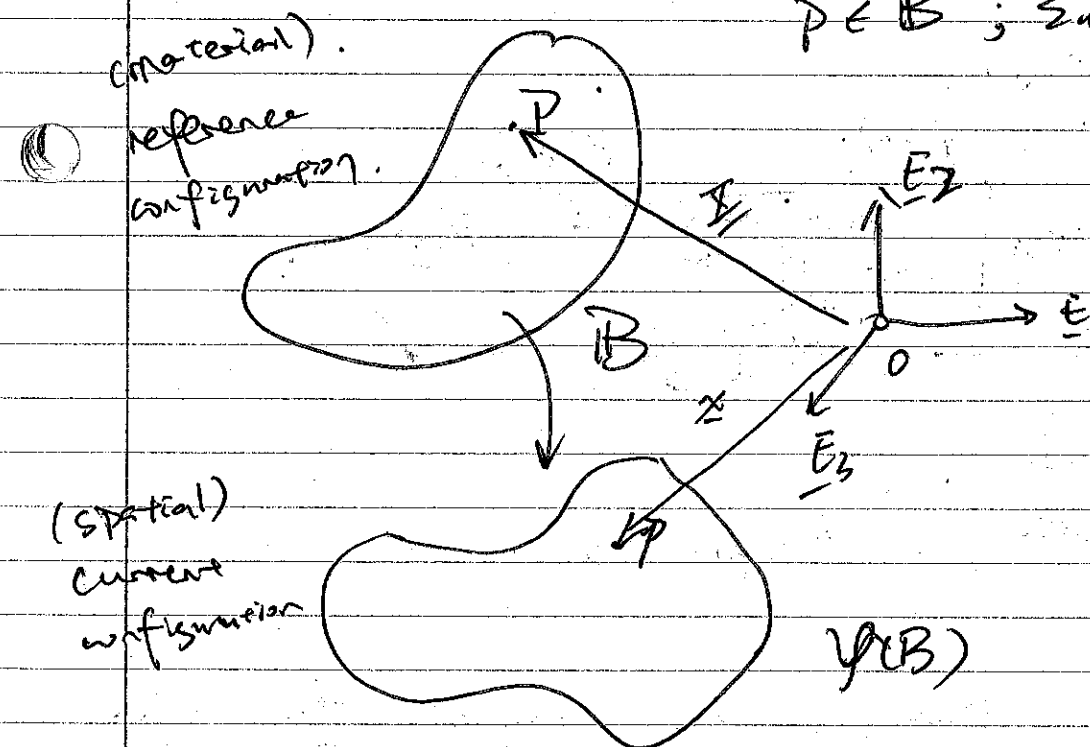
$$\oint_C \underline{a} \cdot d\underline{I} = \int_S \operatorname{curl} \underline{a} \cdot d\underline{S}$$

$$\sum_{\text{de loop}} \underline{a} \cdot d\underline{I} = (\nabla \times \underline{a}) \cdot d\underline{S}$$

Lecture 2. Continuum Mechanics - important

- Kinematics (motion)
 - Strain measures.
 - vector & tensor transformation
 - Balance equations.
 - Stress measures.
 - Constitutive relations.
- this.

$p \in B$; Euclidean space \mathbb{R}^3

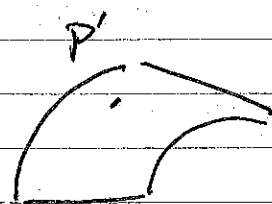
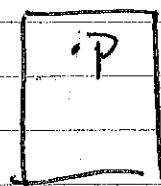


configuration $\varphi: B \rightarrow \mathbb{R}^3$

$$\underline{x} = \varphi(\underline{X}, t)$$

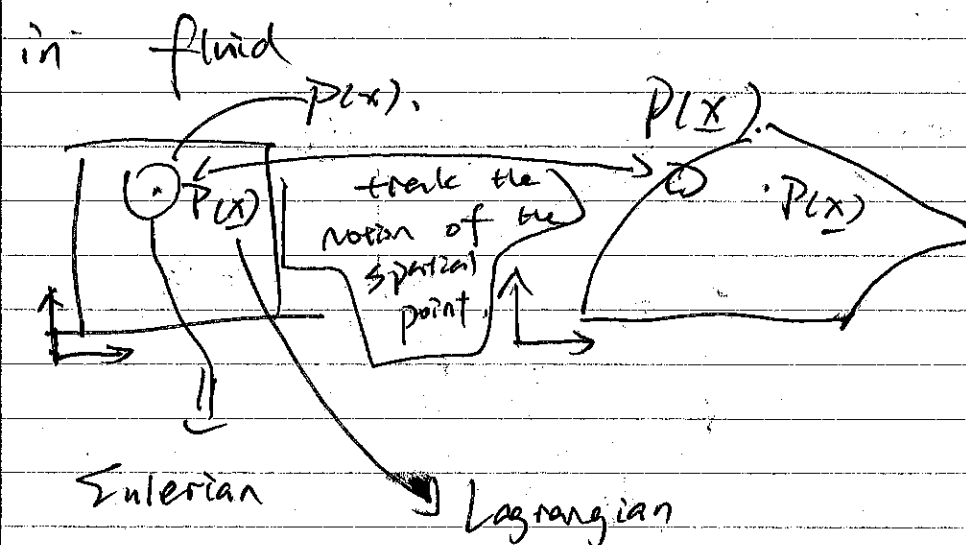
$$\underline{X} =$$

$\underline{x}, \underline{X}$ - positions in \mathbb{R}^3
w/ origin 0

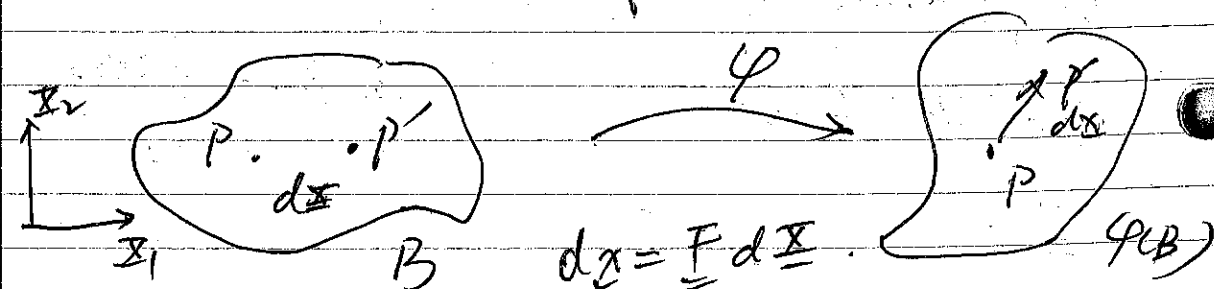


$$\underline{x} = \underline{x}_0 + \underline{\epsilon}$$

in linear FEM - Lagrangian
but does not make a diff.



- How line elements mapped from reference to current config.



$$d\underline{x} = \underline{F} d\underline{X}$$

$$\underline{F} = \frac{\partial \underline{x}}{\partial \underline{X}} \rightarrow \frac{\partial x_i}{\partial X_A} = F_{iA}$$

$$\underline{F} = \text{Grad } \varphi(\underline{X}, t) \rightarrow \underline{x} = \varphi(\underline{X})$$

\underline{F} tensor vs. matrix

preserves basis properties

numerical for n.

$$\underline{A} = \underline{a} \otimes \underline{b}$$

$$\underline{a} = a_i \underline{e}_i$$

$$\underline{b} = b_j \underline{e}_j$$

$$[\underline{A}] = \begin{bmatrix} \dots \end{bmatrix}$$

$$dx_1 = F_{11} dX_1 + F_{12} dX_2 + F_{13} dX_3$$

$$[\underline{F}] = \begin{bmatrix} \frac{\partial x_1}{\partial X_1} & \frac{\partial x_1}{\partial X_2} & \frac{\partial x_1}{\partial X_3} \\ \vdots & \vdots & \vdots \\ \frac{\partial x_3}{\partial X_1} & \frac{\partial x_3}{\partial X_2} & \frac{\partial x_3}{\partial X_3} \end{bmatrix}$$

(continued)

Kinematics - deformation gradients

\underline{F} has to be invertible

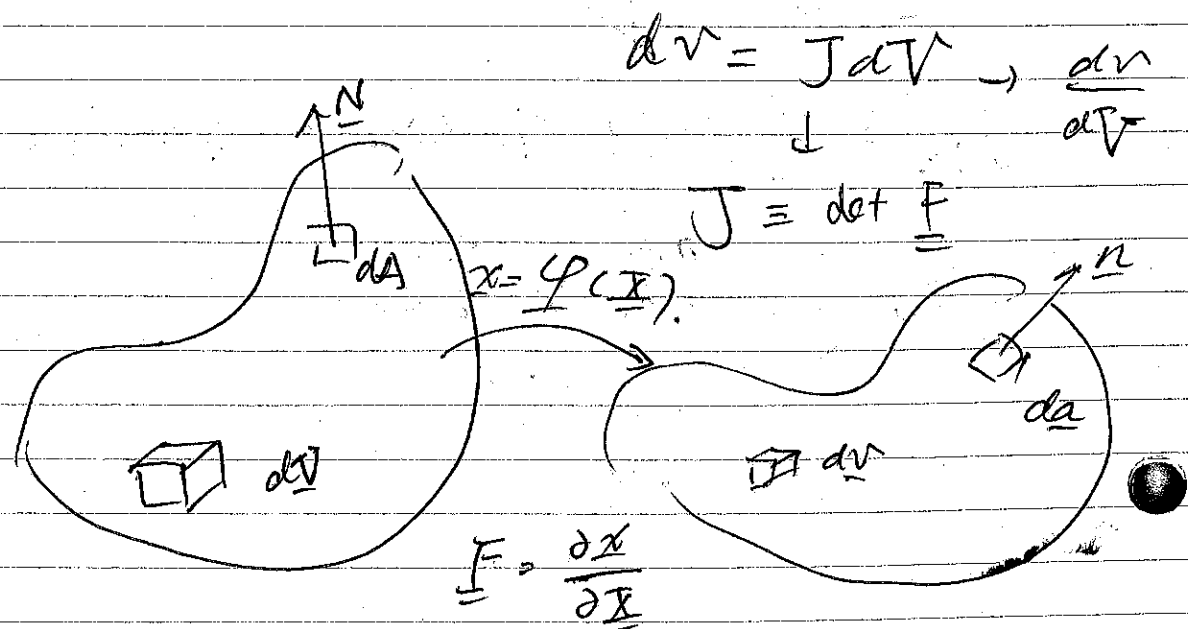
$$d\underline{X} = \underline{F}^{-1} d\underline{x}$$

$$\underline{F}^{-1} = (F^{-1})_A \underline{E}_A \otimes \underline{E}_i$$

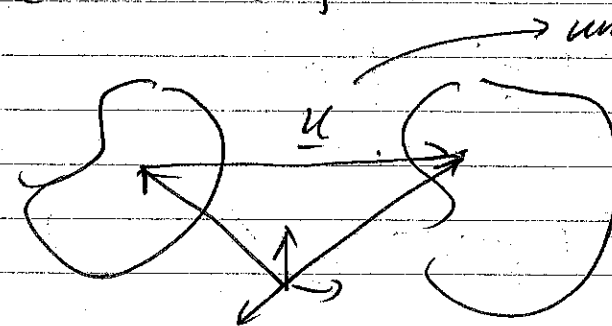
$$\hookrightarrow \frac{\partial \underline{x}_A}{\partial x_i}$$

* Nanson's formula. Capture transformation of surface element in ref. config. dA

$$d\underline{a} = \underline{n} d\underline{a} = J \underline{F}^{-T} \underline{N} dA = J \underline{F}^{-T} d\underline{A}$$



- Define the displacement unknown.



$$\underline{u}(\underline{X}, t) = \underline{\varphi}(\underline{X}, t) - \underline{X} = \underline{x} - \underline{X}$$

$$\underline{F} = \text{Grad} [\underline{X} + \underline{u}(\underline{X}, t)]$$

$$= \underline{I} + \text{Grad} \underline{u} = \underline{I} + \underline{H}$$

displacement gradient

$$\text{Grad} = \nabla_{\underline{X}}$$

- material gradient operation

$$\text{grad} = \nabla_{\underline{x}}$$

- spatial gradient operation

$$\underline{I} = \delta_{ij} \underline{E}_i \otimes \underline{E}_j$$

- Strain measures

Green - Lagrange strain tensor

Right Cauchy-Green strain tensor

$$\underline{E} \equiv \frac{1}{2} (\underline{F}^T \underline{F} - \underline{I}) = \frac{1}{2} (\underline{C} - \underline{I})$$

high-order terms
get rid of

$$\underline{\underline{E}} = \frac{1}{2} (\underline{\underline{H}}^T + \underline{\underline{H}} + \underline{\underline{H}}^T \underline{\underline{H}})$$

$$\underline{\underline{E}} = \frac{1}{2} (\underline{\underline{H}} + \underline{\underline{H}}^T) = \frac{1}{2} (\underline{\underline{U}})$$

Define rotation $\underline{\underline{R}}$ ($\underline{\underline{R}}^{-1} = \underline{\underline{R}}^T$). & corresponding symmetric stretch tensors

$$\underline{\underline{U}} \text{ \& \; } \underline{\underline{V}} : \underline{\underline{F}} = \underline{\underline{R}} \underline{\underline{U}} = \underline{\underline{V}} \underline{\underline{R}}$$

$$\begin{cases} F_{iB} \underline{e}_i \otimes \underline{e}_B = (R_{iA} \underline{e}_i \otimes \underline{e}_A) (U_{cB} \underline{e}_c \otimes \underline{e}_B) \\ F_{iB} \underline{e}_i \otimes \underline{e}_B = (V_{ik} \underline{e}_i \otimes \underline{e}_k) (R_{mB} \underline{e}_m \otimes \underline{e}_B) \end{cases}$$

generalized strain measure:

$$\underline{\underline{E}}^\alpha = \frac{1}{\alpha} (\underline{\underline{U}}^\alpha - \underline{\underline{I}}) \text{ \& \; } \underline{\underline{e}} = \frac{1}{\alpha} (\underline{\underline{V}}^\alpha - \underline{\underline{I}}).$$

$$\underline{\underline{E}} = \frac{1}{2} (\underline{\underline{U}}^2 - \underline{\underline{I}})$$

$$\underline{\underline{U}} = \sum_i \lambda_i \underline{N}_i \otimes \underline{N}_i ; \underline{\underline{V}} = \sum_i \lambda_i \underline{n}_i \otimes \underline{n}_i$$

$$\underline{n}_i = \underline{\underline{R}} \underline{N}_i$$

- Strain measures.

Almansi strain tensor.

$$\underline{\underline{e}} = \underline{\underline{e}}^{(-v)} = \frac{1}{v} (\underline{\underline{I}} - \underline{\underline{V}}^{-v}) = \frac{1}{v} (\underline{\underline{I}} - \underline{\underline{b}}^{-1}) \\ = \frac{1}{v} (\underline{\underline{I}} - \underline{\underline{F}}^T \underline{\underline{F}}^{-1}).$$

$$\underline{\underline{b}} \equiv \underline{\underline{F}} \underline{\underline{F}}^T = \underline{\underline{V}} \underline{\underline{R}} \underline{\underline{R}}^T \underline{\underline{V}}^T = \underline{\underline{V}}^2.$$

* pull back
push forward

$$G(\underline{\underline{X}}) = g(\underline{\underline{x}}).$$

$$\text{Grad } G = \underline{\underline{F}}^T \text{grad } g \Rightarrow \frac{\partial G}{\partial X_A} = \frac{\partial g}{\partial x_i} \frac{\partial x_i}{\partial X_A}$$

$$\text{grad } g = \underline{\underline{F}}^{-T} \text{Grad } G.$$

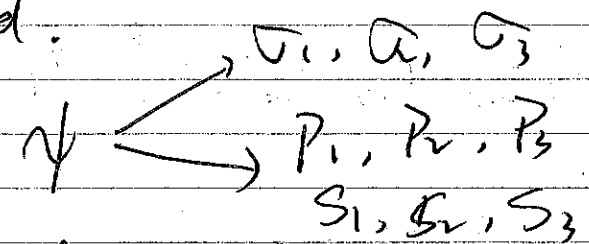
$$\frac{\partial g}{\partial x_i} = \frac{\partial g}{\partial X_A} \frac{\partial X_A}{\partial x_i}$$

pull back
+
push forward

Vector & tensor transformation:

HW continued.

Prob. 9.



Strain Energy function $\Psi \rightarrow \lambda_1, \lambda_2, \lambda_3$.

$$\frac{\partial \Psi}{\partial \lambda_i} = \sigma_i \lambda_i, \quad i=1, 2, 3.$$

$$P_i = \frac{\partial \Psi}{\partial \lambda_i} \quad \sigma_i = \lambda_i P_i.$$

$$\underline{\underline{S}} = 2\rho_0 \frac{\partial \Psi(\underline{\underline{C}})}{\partial \underline{\underline{C}}}$$

$$\underline{\underline{S}} = \underline{\underline{F}}^{-1} \underline{\underline{P}} = J \underline{\underline{F}}^{-1} \underline{\underline{Q}} \underline{\underline{F}}^{-T}.$$

Ψ — incompressible isotropic hyperelastic.

i.e. o. principal stretches.

$$\Psi = \frac{1}{2} k (\lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3)$$

$$+ \frac{k}{\beta} (\lambda_1 \lambda_2 \lambda_3)^\beta$$

||
3 Cauchy stress.

$$\begin{cases} \sigma_1 = k \lambda_1 (\lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3) + (\lambda_1 \lambda_2 \lambda_3)^\beta k \beta \lambda_1 \\ \sigma_2 = k \lambda_2 (\quad \quad \quad) + \quad \quad \quad \lambda_2 \\ \sigma_3 = k \lambda_3 (\quad \quad \quad) + \quad \quad \quad \lambda_3 \end{cases}$$

$$\Psi = (I_1, I_2, I_3) = \frac{\mu}{2} (I_1 - 3) + \frac{\lambda}{2} (I_1 - 3)^2$$

$$\Psi = \frac{\mu}{2} (I_1 - 3)$$

$$= \frac{\mu}{2} (\lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3)$$

↓

$$\sigma_1, \sigma_2, \sigma_3 = \dots$$

$$\underline{P} = -p \underline{F}^{-T} + 2 \left[\left(\frac{\partial \Psi}{\partial I_1} + I_1 \frac{\partial \Psi}{\partial I_2} \right) \underline{F} - \frac{\partial \Psi}{\partial I_2} \underline{F} \underline{F} \right]$$

Prob 10

$$\chi_1 = \lambda_1 \underline{I}_1, \quad \chi_2 = \lambda_2 \underline{I}_2, \quad \chi_3 = \frac{1}{\lambda_1 \lambda_2} \underline{I}_3$$

★ Membrane theory:

$$\text{Ogden's theory: } \sigma = \mu (\lambda^n - 1)$$

$$W = \sum_i \frac{\alpha_i}{n_i} \left[(\lambda_1^2 + \lambda_2^2 + \frac{1}{\lambda_1^2} + \frac{1}{\lambda_2^2})^{\frac{n_i}{2}} - 2 \right]$$

Strain energy (Wiki) $W(\lambda_1, \lambda_2) = \sum_i \frac{\alpha_i}{n_i} (\lambda_1^{\alpha_i} + \lambda_2^{\alpha_i} + \lambda_1^{-\alpha_i} \lambda_2^{-\alpha_i} - 3)$

We know λ_1 & λ_2 & $\det \underline{F} = 1$

HW note.

Prob 7: reverse to get $\underline{X}(\underline{x})$.

$$\begin{cases} \chi_1 = e^t \underline{I}_1 - e^{-t} \underline{I}_2 \\ \chi_2 = e^t \underline{I}_1 + e^{-t} \underline{I}_2 \\ \chi_3 = \underline{I}_3 \end{cases}$$

$$\underline{u} = \underline{x} - \underline{X}$$

$$\underline{F} = \frac{\partial \underline{x}}{\partial \underline{X}} = \frac{\partial (\underline{u} + \underline{X})}{\partial \underline{X}}$$

$$\begin{cases} \chi_1 + \chi_2 = 2e^t \underline{I}_1 \rightarrow \underline{I}_1 = \frac{1}{2e^t} (\chi_1 + \chi_2) \\ \chi_2 - \chi_1 = 2e^{-t} \underline{I}_2 \rightarrow \underline{I}_2 = \frac{1}{2e^{-t}} (\chi_2 - \chi_1) \\ \underline{I}_3 = \chi_3 \end{cases}$$

in the material description:

$$\underline{A}(\underline{X}, t) = \frac{\partial \underline{V}(\underline{X}, t)}{\partial t} = \begin{cases} \frac{\partial \chi_1}{\partial t} = e^t \underline{I}_1 - e^{-t} \underline{I}_2 \\ \frac{\partial \chi_2}{\partial t} = e^t \underline{I}_1 + e^{-t} \underline{I}_2 \\ \frac{\partial \chi_3}{\partial t} = 0 \end{cases}$$

$$\underline{V}(\underline{X}, t) = \frac{\partial \underline{x}(\underline{X}, t)}{\partial t} = \begin{cases} \frac{\partial \chi_1}{\partial t} = e^t \underline{I}_1 + e^{-t} \underline{I}_2 \\ \frac{\partial \chi_2}{\partial t} = e^t \underline{I}_1 - e^{-t} \underline{I}_2 \\ \frac{\partial \chi_3}{\partial t} = 0 \end{cases}$$

$$\frac{1}{4e^{2t}} \frac{1}{e^t} = \begin{bmatrix} e^t \underline{I}_1 + e^{-t} \underline{I}_2 \\ e^t \underline{I}_1 - e^{-t} \underline{I}_2 \\ 0 \end{bmatrix}$$

Prob. 8:

$$d\underline{u} = (\underline{F} - \underline{I}) d\underline{x}$$

$$\frac{d\underline{u}}{d\underline{x}} = \underline{F} - \underline{I}$$

$$\frac{d\underline{u}}{d\underline{x}} + \underline{I} = \underline{F}$$

$$\underline{x} - \underline{x} = \underline{u}$$

$$\frac{\partial \underline{x}}{\partial \underline{x}} - \frac{\partial \underline{x}}{\partial \underline{x}} = \frac{\partial \underline{u}}{\partial \underline{x}}$$

$$\underline{I} - \underline{F}^{-1} = \frac{\partial \underline{u}}{\partial \underline{x}}$$

$$\underline{I} - \frac{\partial \underline{u}}{\partial \underline{x}} = \underline{F}^{-1}$$

$$\underline{F}^{-1} = \begin{bmatrix} 1 & -\frac{1}{4} & 0 \\ 1 & 2 & 0 \\ 0 & 0 & -3 \end{bmatrix}$$

$$\underline{F}^{-1} = \begin{bmatrix} 0 & -\frac{1}{4} & 0 \\ 1 & -1 & 0 \\ 0 & 0 & 4 \end{bmatrix}$$

$$\underline{F} = \begin{bmatrix} 0 & -\frac{1}{4} & 0 \\ -\frac{1}{4} & -\frac{1}{4} & 0 \\ 0 & 0 & \frac{1}{4} \end{bmatrix}$$

$$\underline{F} = \frac{1}{\det(\underline{F})} \text{adj}(\underline{F})$$

$$\text{adj}(\underline{F}^{-1}) = \begin{bmatrix} 0 & \frac{1}{4} & 0 \\ 1 & 1 & 0 \\ 0 & 0 & 4 \end{bmatrix}$$

Prob. 9 & 10.

$$\frac{\partial \psi}{\partial \underline{c}} = \sum_i^3 \frac{\partial \psi}{\partial \lambda_i^2} \frac{\partial \lambda_i^2}{\partial \underline{c}}$$

$$\frac{\partial \lambda_i^2}{\partial \underline{c}} = \underline{n}_i \underline{n}_i$$

$$\hookrightarrow \frac{\partial \psi}{\partial \underline{c}} = \sum_i^3 \frac{\partial \psi}{\partial \lambda_i^2} \underline{n}_i \underline{n}_i$$

$$\underline{S} = 2 \frac{\partial \psi}{\partial \underline{c}} = \sum_i^3 \frac{2 \frac{\partial \psi}{\partial \lambda_i^2} \underline{n}_i \underline{n}_i}{\frac{\partial (\lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3)}{\partial \lambda_i^2}} \underline{n}_i \underline{n}_i$$

$$= \frac{2 \lambda_1 \underline{n}_1}{2 \cdot \underline{n}_1} + \frac{2 \lambda_2 \underline{n}_2}{2 \cdot \underline{n}_2} + \frac{2 \lambda_3 \underline{n}_3}{2 \cdot \underline{n}_3}$$

$$\underline{I} = \underline{F} \underline{S}$$

$$\underline{n}(\underline{n}_1 \underline{n}_1 + \underline{n}_2 \underline{n}_2 + \underline{n}_3 \underline{n}_3)$$

$$\hookrightarrow \underline{F} \underline{N}_i = \underline{\lambda}_i \underline{n}_i$$

$$\underline{F} \underline{N}_i = \underline{\lambda}_i \underline{n}_i$$

$$\underline{P} = \underline{F} \sum_i^3 \frac{\partial \psi}{\partial \lambda_i^2} \underline{N}_i \otimes \underline{N}_i = \sum_i^3 \frac{\partial \psi}{\partial \lambda_i^2} \underline{N}_i \otimes \underline{N}_i$$

$$\underline{U} = \underline{J}^{-1} \underline{F} \underline{P} = \frac{1}{\lambda_1 \lambda_2 \lambda_3} \underline{n}(\lambda_1 \underline{N}_1 \underline{N}_1 + \lambda_2 \underline{N}_2 \underline{N}_2 + \lambda_3 \underline{N}_3 \underline{N}_3)$$

Prob 10.

Ogden model: plane stress $\sigma_3 = 0$

$$y = \sum_i^3 \frac{2w}{\alpha_i^2} (\bar{\gamma}_1^{\alpha_i} + \bar{\gamma}_2^{\alpha_i} + \gamma_3^{\alpha_i} - 3) + \frac{K_1}{2} (J-1)^2$$

$$\frac{\partial \psi}{\partial \lambda_i} = \sum_i^3 \frac{2 \mu_i}{\left(\frac{\lambda_i}{J^{1/3}}\right)^{4/3}}$$

↳ const. $\alpha_i \approx 1$

$$\frac{2n_i}{\alpha_i^2} \left(\frac{1}{J^{1/3}} \right)^{\alpha_i} \alpha_i \geq \alpha_i - 1$$

$$= \frac{2^{n_i}}{\alpha_i} \left(\frac{1}{j^{1/3}} \right)^{\alpha_i} \gamma_i^{\alpha_i-1}$$

$$F = \frac{\partial \mathcal{L}}{\partial \underline{x}} = \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \lambda_2 & 0 \\ 0 & 0 & \lambda_1 \lambda_2 \end{bmatrix}$$

$$J=1$$

Week 3 - Lesson 1.

next form

Tuttes maktz

EC.

$w = \{w \mid w \in H', w=0 \text{ on } \Gamma_g\}$

integrability \Rightarrow

$u \Rightarrow u = \{u \mid u \in H', u=g \text{ on } \Gamma_g\}$

$\int_{\Omega} |\nabla w|^2 dV < \infty$

does not blow up

* Cannot prescribe both force & displacement

"Spring"-type BCs - Robin

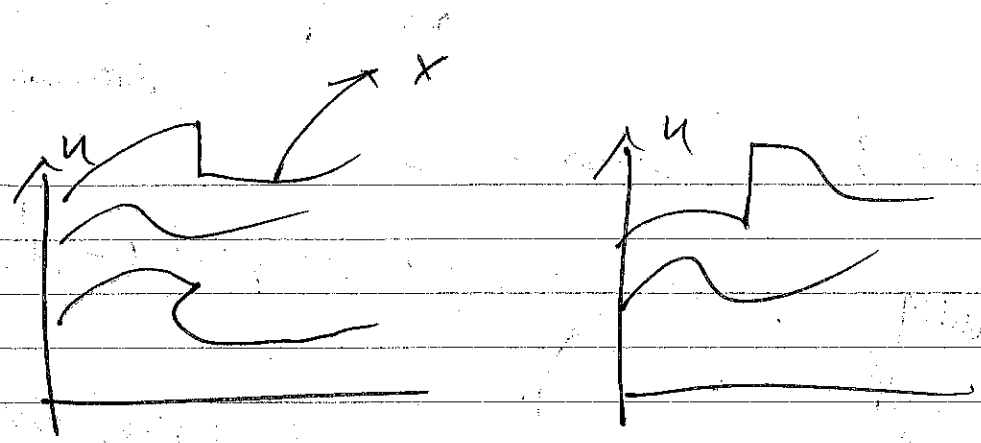
essential vs. natural

Dinichlet vs. Neumann

$$u = g \quad - g \cdot n_i = h$$

1st: multiply both sides by test function

$\Gamma = \nabla g \cup \Gamma_h$
 $\oint_{\Gamma} \vec{q} \cdot \vec{n} = f$
 $\oint_{\Omega} w \operatorname{div} \vec{q} = w f$
 $\int_{\Omega} w \operatorname{div} \vec{q} \, dV = \int_{\Omega} w f \, dV$
 $\int_{\Omega} w \vec{q} \cdot \vec{n} \, dS - \int_{\Omega} \vec{q} \cdot \vec{g} \operatorname{grad} w \, dV = \int_{\Omega} w f \, dV$



\therefore we find u has to be ~~smooth~~ continuous

$C^0 \rightarrow$ zeroth derivative is continuous

$C^1 \rightarrow$ first derivative is continuous

$C^n \rightarrow$ n^{th} derivative is continuous.

1) \star Strong form: point-wise
weak form: on an average sense.

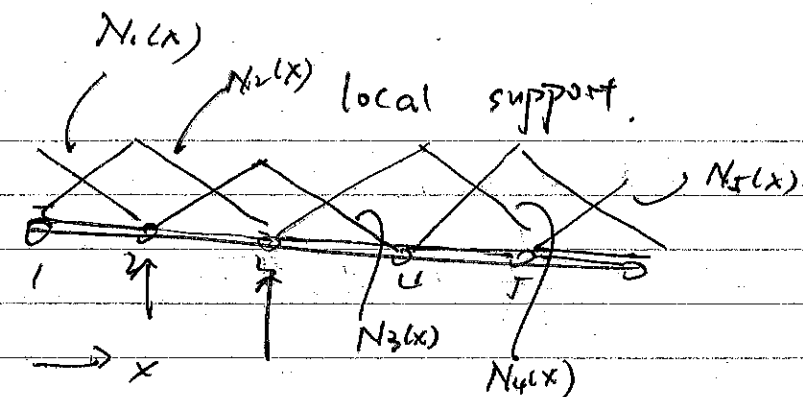
Order
2) Strong form: $a_{ij} = k_{ij} u_{,ij}$ - 2nd order

weak form: - 1st order deriv.

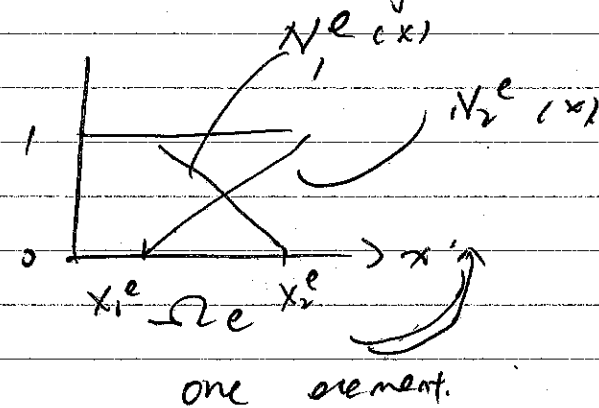
\hookrightarrow symmetry between w & u

- Shape function example:

1D domain:

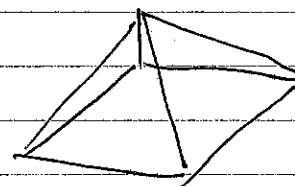
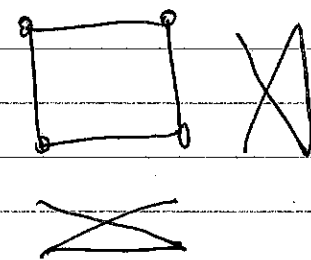
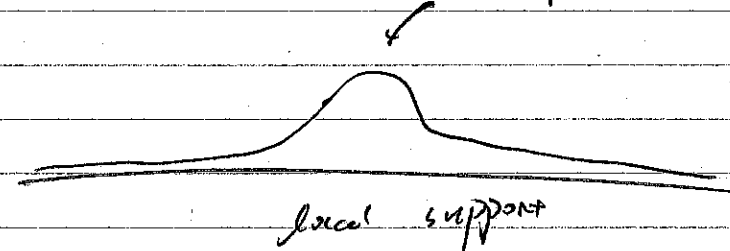


all the shape functions look the same.



- linear shape functions.

"bubble" function



$$\Rightarrow (W^h, u)_p = \sum_i C_A(N_A, u),$$

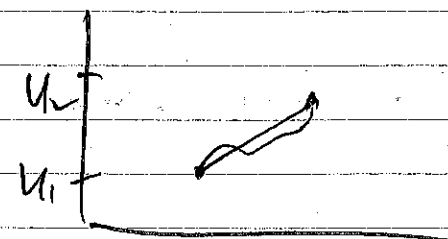
many integrals.

$$= \int_{\Gamma} N_1 C_1 ds + \int_{\Gamma} N_2 C_2 ds + \dots$$

$$\sum_{A \in \Gamma-13} C_A \left\{ \begin{array}{l} \text{ } \end{array} \right\} = 0 \quad \text{we can set}$$

$$u \approx u_1 N_1 + u_2 N_2$$

$$\frac{\partial u}{\partial x} \approx u_1 \frac{\partial N_1}{\partial x} + u_2 \frac{\partial N_2}{\partial x}$$

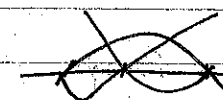


#FEM

Week 3 - Lecture 2.

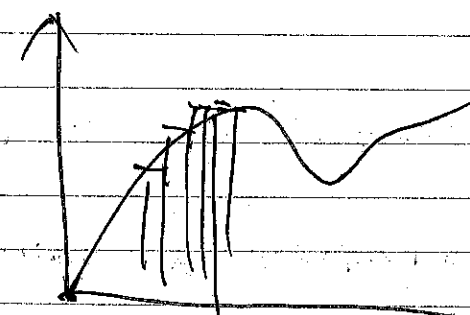
Galerkin formulation.

$$[K] \begin{Bmatrix} d_1 \\ d_2 \\ \vdots \\ d_n \end{Bmatrix} = \{F\} \rightarrow \text{solve for } d$$



quadratic

quadratic



Gauss quadrature

sample \rightarrow integration method

\nearrow Same shape function
 Bubbiu - Galerkin \rightarrow same s.f.
 Petrov - Galerkin \rightarrow different s.f.
 \hookrightarrow subg.

how to solve PDE \rightarrow tune the shape functions

Galerkin method vs. Rayleigh Ritz method

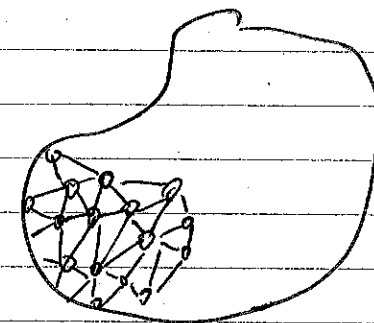
\Downarrow
 Shape function
has to be global

\swarrow C_0 : derivatives have a jump.

① what order derivatives in weak form

② what order continuous is required.

Week 4. Lecture 1.



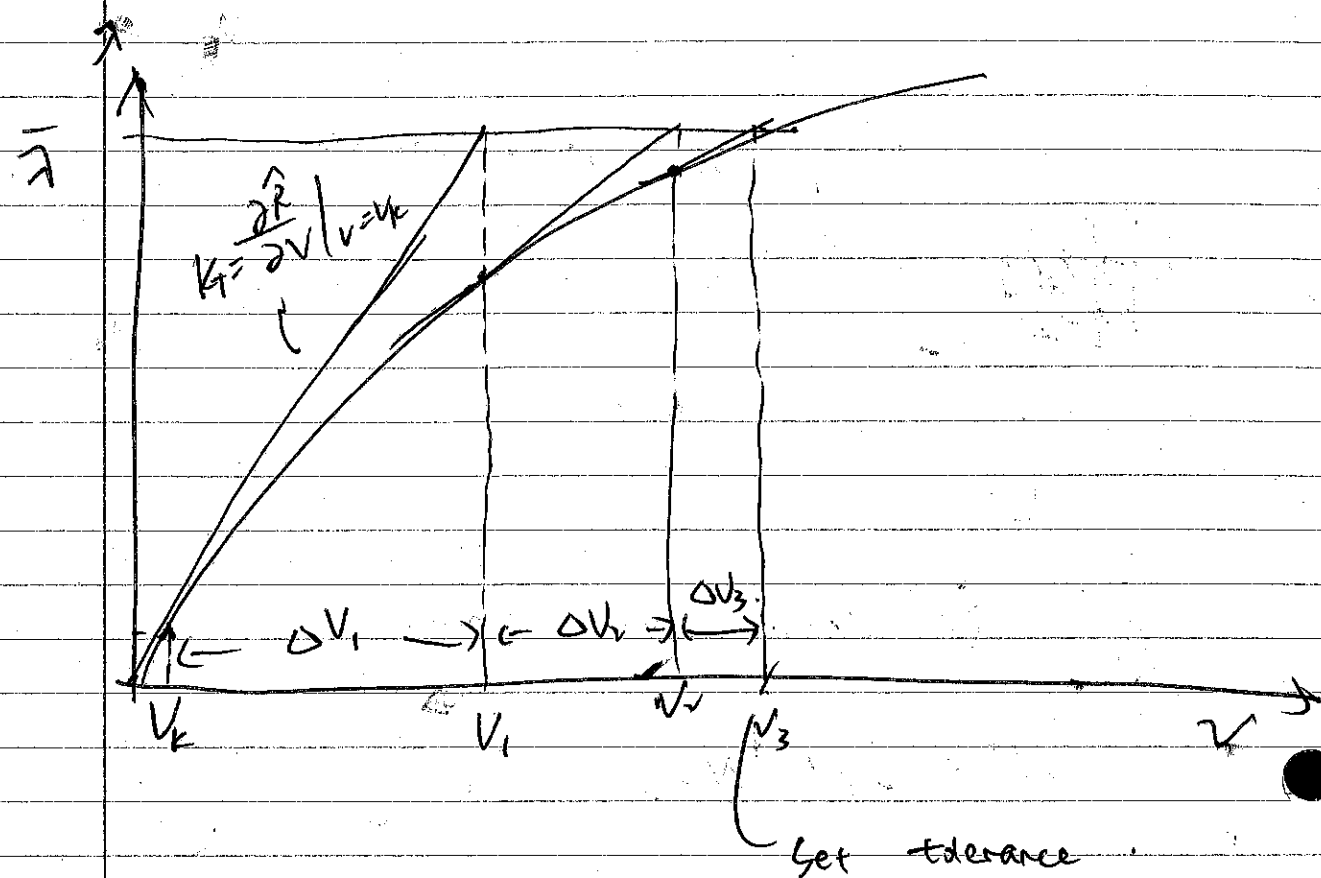
$u, u^h \rightarrow$ ele. size

$\hookrightarrow L^2$

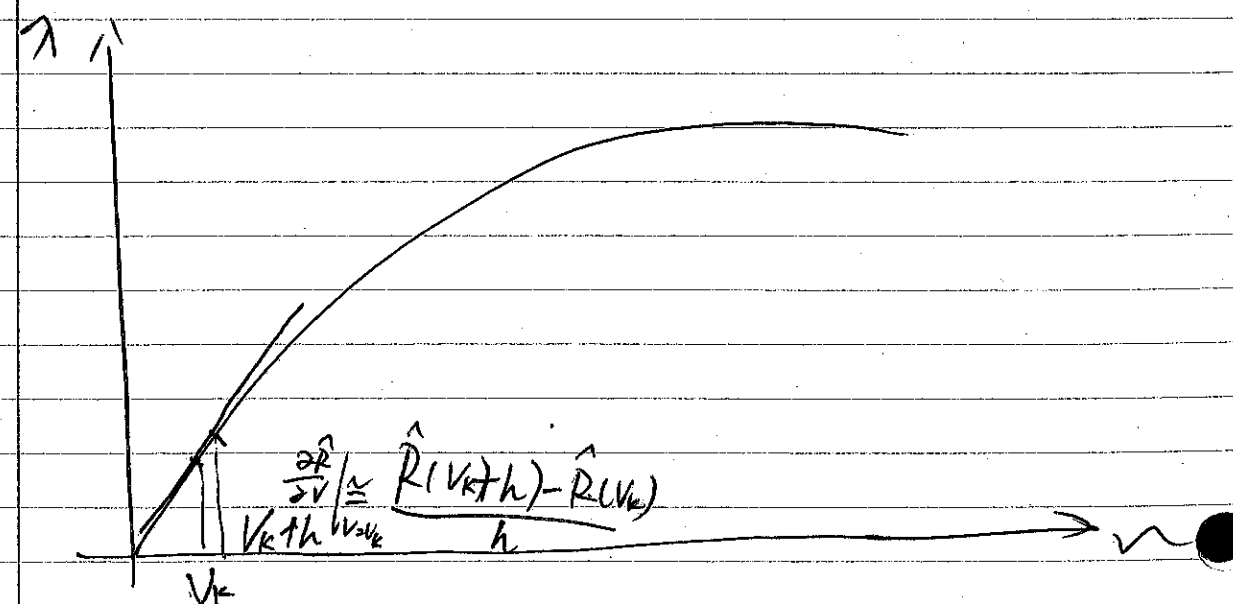
eg. 1D linear shape func.

$$N^e = [N_1, N_2] \quad L^e = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

Week 5, Lecture 1

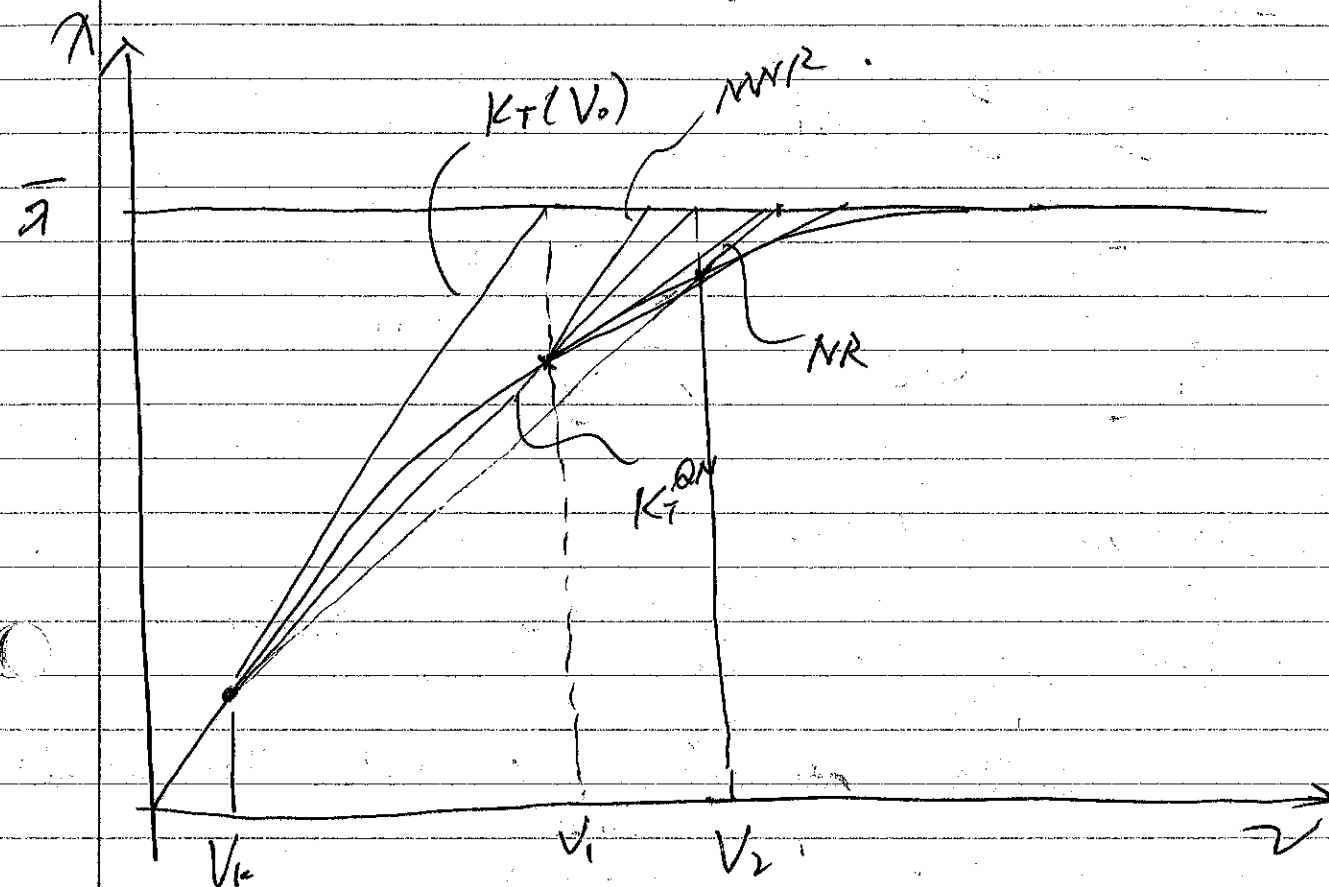


"Quadratic Convergence"



Lecture 2

Comparison between 3 methods



HW2 2

Newton - Raphson

$$\underline{G} = \underline{R} - \lambda \underline{P}$$

$$\begin{bmatrix} G_1 \\ G_2 \end{bmatrix} = \begin{bmatrix} R_1 \\ R_2 \end{bmatrix} - \lambda \begin{bmatrix} P_1 \\ P_2 \end{bmatrix} \rightarrow 0$$

$$V_0 = V_k \quad (ICs).$$

For λ_i in λ ← initialize

while $\Sigma \geq 1e-4$ and iter ≤ 15 .

$$\underline{G} + \underline{D} \underline{G} \Delta \underline{V}$$

$$\begin{bmatrix} G_1 \\ G_2 \end{bmatrix} + \begin{bmatrix} D_1 G_1 & D_1 G_2 \\ D_2 G_1 & D_2 G_2 \end{bmatrix} \begin{bmatrix} \Delta V_1 \\ \Delta V_2 \end{bmatrix}$$

equiv.

$$\underline{K}_T \Delta \underline{V} = -\underline{G} \quad \dots (1)$$

$$\underline{V} = \underline{V} + \Delta \underline{V} \quad \dots (2)$$

$$\|\underline{G}\| = \sqrt{G[0]^2 + G[1]^2} \quad (3)$$

for $j = [0, 1, 2, \dots, 40]$

$$\Delta \lambda = 0.25$$

$$\lambda = \Delta \lambda * j$$

$$\underline{G}_0 = \underline{K}_T \Delta \underline{V}_0$$

while $\Sigma \geq 1e-4$ & iter ≤ 15 .

$$\underline{G} = \underline{R} - \lambda \underline{P}$$

$$\underline{K}_T = [\quad]$$

$$\underline{V} = \underline{V} + \Delta \underline{V}$$

$$\|\underline{G}\| = \sqrt{\dots}$$

$$\underline{G} = - \begin{bmatrix} 0.6V_1 \Delta V_1 + 6\Delta V_1 - 2xV_2 \Delta V_2 \\ -\Delta V_1 + \Delta V_2 \end{bmatrix}$$

$$\underline{K}_T = \begin{bmatrix} 0.6V_1 + 6 & -2xV_2 \\ -2xV_2 & 1 \end{bmatrix} \begin{bmatrix} \Delta V_1 \\ \Delta V_2 \end{bmatrix}$$

Newton-Raphson implementation.

Initialization

$$\text{for } \lambda \begin{cases} \text{while} \\ V_0 = V_k = \begin{bmatrix} \cdot \\ \cdot \end{bmatrix} \end{cases}$$

$i \leq 15$

Compute G

$$G = R - \lambda P \Rightarrow \begin{bmatrix} G_1 \\ G_2 \end{bmatrix} = \begin{bmatrix} R_1 \\ R_2 \end{bmatrix} - \lambda \begin{bmatrix} P_1 \\ P_2 \end{bmatrix}$$

Compute K_T

$$K_T = DG = \begin{bmatrix} \lambda G_1 & \lambda G_2 \\ D_1 G_1 & D_2 G_2 \end{bmatrix}$$

Compute ΔV_{i+1}

$$\Delta V_{i+1} = \frac{G}{K_T} = \frac{G}{K_T} - \text{or } K_T^{-1} G$$

Compute new V

$$V_{i+1} = V_i + \Delta V_{i+1}$$

$i = i + 1$

Compute convergence

$$E = \sqrt{G[0]^2 + G[1]^2}$$

How to solve I.C.S?

$$\begin{bmatrix} 0.2 V_1^3 - \lambda V_2^2 + 6 V_1 \\ V_2 - V_1 \end{bmatrix} = 0$$

$$0.2 V_1^3 - \lambda V_2^2 + 6 V_1$$

HW Line Search Method

\downarrow

$$g(\alpha_i) = \Delta V_{i+1}^T \left[\frac{\text{Select } \alpha_i}{G(V_i + \alpha_i \Delta V_{i+1}, \lambda)} \right] = 0$$

$$G = \begin{bmatrix} 0.2 V_1^3 - \lambda V_2^2 + 6 V_1 \\ V_2 - V_1 \end{bmatrix} - \lambda_i \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

$$g(\alpha_i) = \begin{bmatrix} \Delta V_{i+1}[0] \\ \Delta V_{i+1}[1] \end{bmatrix} \left\{ \begin{bmatrix} 0.2 (V_1 + \alpha_i \Delta V_1)^3 - \lambda (V_2 + \alpha_i \Delta V_2)^2 + 6 (V_1 + \alpha_i \Delta V_1) \\ V_2 + \alpha_i \Delta V_2 - V_1 - \alpha_i \Delta V_1 \end{bmatrix} - \lambda_i \begin{bmatrix} 1 \\ 0 \end{bmatrix} \right\}$$

to solve for α_{i+1} next α :

$$\alpha_i^{k+1} = \alpha_i^k - g(\alpha_i^k) \left[\frac{\alpha_i^k - \alpha_i^{k-1}}{g(\alpha_i^k) - g(\alpha_i^{k-1})} \right]$$

BFGS Method

Initialization

$$V_0 = V_k$$

Compute G

... (# same)

Compute K

... (# same).

$$\Rightarrow K(0).$$

For loop ...

• compute G

• compute $H \approx H_k$

$$\underline{W}_i = \underline{V}_i - \underline{V}_{i-1}$$

$$[\] = [\] - [\]$$

$$\underline{g}_i = \underline{G}_i - \underline{G}_{i-1}$$

$$[\] = [\] - [\]$$

$$\underline{a} = \frac{1}{\text{np.transpose}(\underline{g}_i) \cdot \underline{W}_i} \cdot \underline{W}_i$$

$$\underline{b}_i = - \left\{ \underline{g}_i - \left[\frac{-\underline{W}_i^T \underline{g}_i}{\underline{W}_i^T \underline{G}_{i-1}} \right]^{\frac{1}{2}} \underline{G}_{i-1} \right\}$$

BFGS

def: compute $V(H, G, V)$

- $\Delta V = -HG$
- $V = V + \Delta V$
- $\rightarrow V_{i-1}, V_i$

def: compute $G(R, \lambda(P), V)$

- $G_{\text{new}} \leftarrow G = R - \lambda P$
- $G \leftarrow G_{\text{new}} = R_{\text{new}} - \lambda P$
- $\rightarrow G_{i-1}, G_i$

$$\underline{G}_i, \underline{G}_{i-1} \leftarrow$$

$$\underline{V}_i, \underline{V}_{i-1} \leftarrow$$

$$\underline{W}_i = \underline{V}_i - \underline{V}_{i-1}$$

$$\underline{g}_i = \underline{G}_i - \underline{G}_{i-1}$$

$$a_i = \frac{1}{\underline{g}_i^T \underline{W}_i \underline{W}_i}$$

$$b_i = - \left\{ \underline{g}_i - \left[\frac{-\underline{W}_i^T \underline{g}_i}{\underline{W}_i^T \underline{G}_{i-1}} \right]^{\frac{1}{2}} \underline{G}_{i-1} \right\}$$

$$\rightarrow H_i = H_i(a_i, b_i, H_{i-1})$$

for BFGS.

we have to solve exactly for

V & V_{prev} and G & G_{prev}
before the ∇ loop:

V_{new} exact solution:

def compute $G_i(\lambda, v)$.

$\rightarrow G_i$

def compute $H(V_i, V_{i-1}, G_i, G_{i-1}, \lambda)$.

$\rightarrow H$

def solve $v(\lambda)$.

"Solve v analytically"

$\rightarrow v$

for $\lambda = [0, 0.25, 0.5, \dots, 1.0]$

$V_{i-1}, V_i, G_{i-1}, G_i$

while not converge:

$H = \text{compute } H$

$\Delta V = H \cdot G_i$

$\rightarrow V_{i+1} = V_i + \Delta V_{i+1}; V_i = V_{i-1}$

Solve for next v :

$$\begin{bmatrix} 0.2V_1^3 - \lambda V_2^2 + 6V_1 \\ V_2 - V_1 \end{bmatrix} - 0.25 \begin{bmatrix} 1 \\ 1 \end{bmatrix} = 0$$

$$0.2V_1^3 - \lambda V_2^2 + 6V_1 - 0.25 = 0$$

Week 6: Lecture 1.

Weak form & variational principles

(Lecture 11).

- finite deformations

↳ geometric nonlinearities

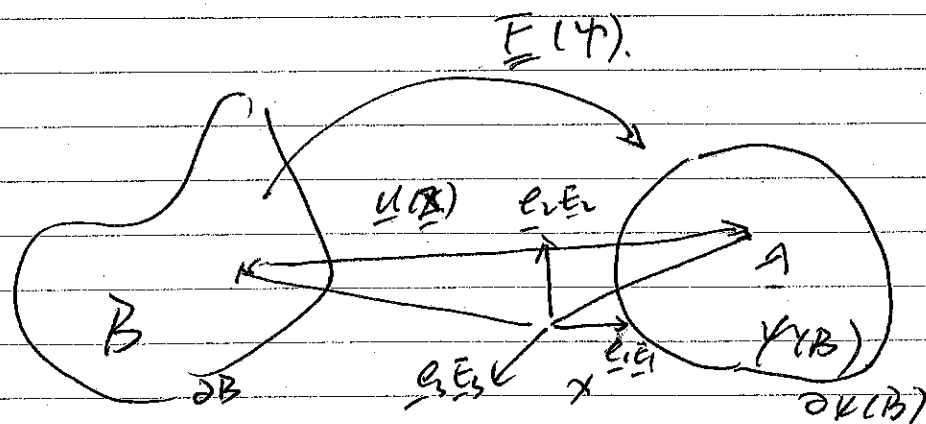
↳ material nonlinearities (const. law)

Strong form

→ Weak form

{ material description
spatial disp.

{ material ~
spat. ~



Strong form: hyperelasticity.

- Kinematics:

$$\underline{x} = \underline{\psi}(\underline{X}) = \underline{u} + \underline{X}$$

reference config.

$$\underline{F} = \text{Grad } \underline{\psi}(\underline{X}, t) = \underline{I} + \text{Grad } \underline{u}$$

- Constitutive law.

$$\underline{P} = \frac{\partial W(\underline{F})}{\partial \underline{F}}$$

fixed.

Equilibrium:

body forces
per unit
ref. vol.

strain energy
function.

$$\text{Div } \underline{P} + \underline{b} = \rho_0 \underline{\dot{v}} \quad \underline{W} = W(I_1, I_2, I_3)$$

principle invariants
of $\underline{C} = \underline{F}^T \underline{F}$.

Material divergence.

"2nd order"

- BCs:

$$\partial B_u \cup \partial B_\sigma = \partial B \quad \underline{u} = \underline{\bar{u}} \quad \text{on } \partial B_u \quad \text{essential}$$

$$\partial B_u \cap \partial B_\sigma = \emptyset \quad \underline{P} \underline{N} = \underline{\bar{t}} \quad \text{on } \partial B_\sigma \quad \text{natural}$$

2.L. strain tensor

- kinematic

$$\underline{\underline{E}} = \frac{1}{2} (\underline{\underline{F}}^T + \underline{\underline{F}} - \underline{\underline{I}}) \quad \underline{\underline{F}} = \underline{\underline{I}} + \text{Grad} \underline{\underline{u}}$$

- const. mod.

$$\underline{\underline{S}} = \frac{\partial W}{\partial \underline{\underline{E}}}$$

- equilibrium

$$\text{Div} (\underline{\underline{F}} \underline{\underline{S}}) + \underline{\underline{b}} = \rho_0 \underline{\underline{v}}$$
↑
reference density

- BCS.

current config.

$\underline{\underline{b}}$ - per unit current volume

ρ - current density

$$\underline{\underline{D}} = \frac{1}{J} \underline{\underline{I}} \leftarrow \text{Kirchhoff stress tensor}$$

lecture 13.

$\underline{\underline{\eta}} \cdot \underline{\underline{f}} \rightarrow \text{scalar}$

$\underline{\underline{\eta}} : \underline{\underline{f}}$

Single contraction

$$\underline{\underline{A}} \underline{\underline{B}} = A_{IK} B_{KJ} \underline{\underline{e}}_I \underline{\underline{e}}_J = M_{IJ}$$

$$\underline{\underline{A}} \cdot \underline{\underline{B}} = A_{IJ} B_{IJ} = \alpha$$

↓ trace of matrix

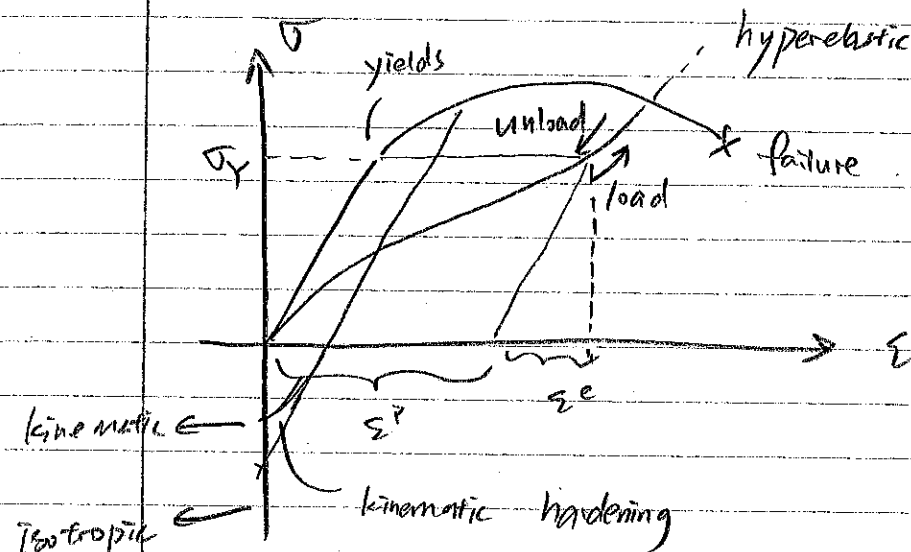
$$\underline{\underline{A}} \underline{\underline{B}} = \underline{\underline{M}}$$

$$\underline{\underline{A}} \cdot \underline{\underline{B}} = \alpha$$

Conclude: Course Notes for FEM

Additional Notes on Plasticity

plasticity vs. hyperelasticity

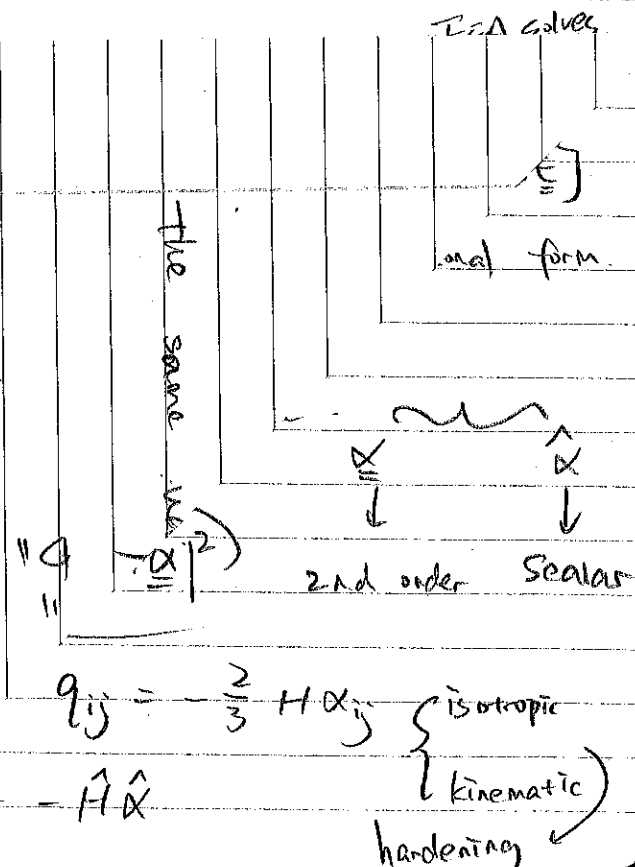


hyperelasticity has a one-on-one mapping between the stress and strains

Hyperelastic: $\rightarrow \underline{\underline{S}}$

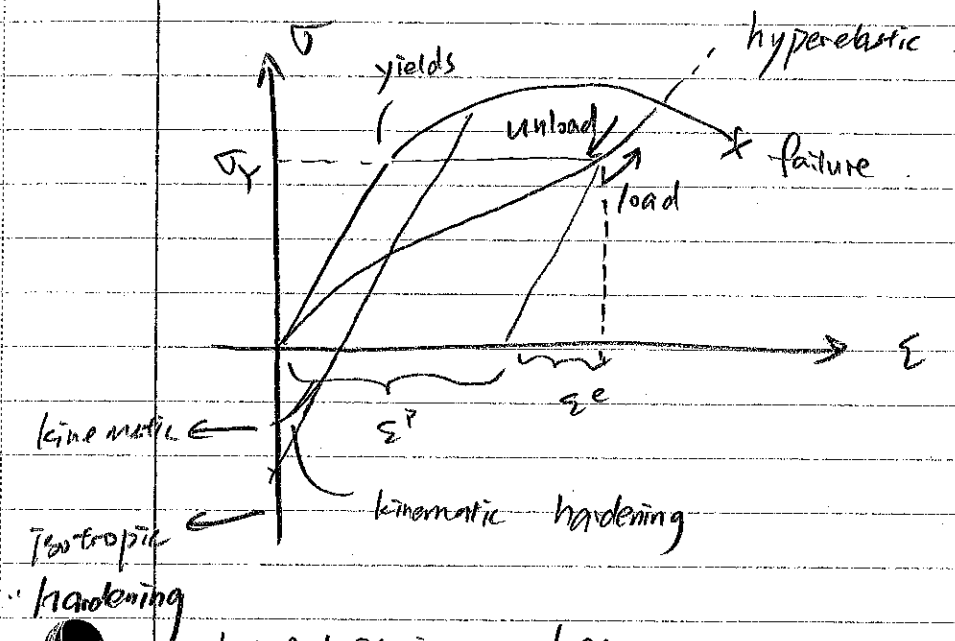
$$G(\bar{\varphi}, \eta) + \Delta C$$

plasticity - easier to
dissipate energy



● # Additional Notes on Plasticity

plasticity vs. hyper-elasticity.



hyperelasticity has a one-on-one mapping between the stress and strains

Hyperelastic: $\rightarrow \underline{\underline{S}} = \gamma \frac{\partial W(\underline{\underline{C}})}{\partial \underline{\underline{C}}} = \gamma(\underline{\underline{C}})$

FEA solves.

$u = \bar{u} + \Delta u$
 \downarrow
 $\underline{\underline{C}}$

$G(\bar{\varphi}, \gamma) + \Delta G \cdot \Delta u \approx 0$

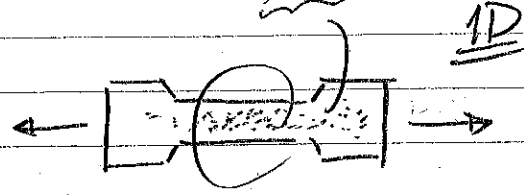
↑
Calculate stresses in this term.

plasticity - easiest version

↓
irreversibility.

dissipate energy to induce plastic deformation.

thinking an experiment
assume macroscopic pos-def



homogeneous stress state

why we like it

Assume small deformations:
or kinematic
"empirical assumption from obs."

$$\underline{\underline{\epsilon}} = \underline{\underline{\epsilon}}^e + \underline{\underline{\epsilon}}^p$$

elastic plastic

(Additive strain decomposition)

deviatoric

$$\underline{\underline{e}} = \underline{\underline{\epsilon}} - \frac{1}{3} \text{tr} \underline{\underline{\epsilon}} \underline{\underline{I}}$$

$$\underline{\underline{S}} = \underline{\underline{\sigma}} - \frac{1}{3} \text{tr} \underline{\underline{\sigma}} \underline{\underline{I}}$$

conserv. angular momentum

the same way we define a strain energy

$$\underline{\underline{\sigma}} = \frac{\partial \psi(\underline{\underline{\epsilon}}^e, \underline{\underline{\alpha}})}{\partial \underline{\underline{\epsilon}}^e}$$

stress & strain → external

$$\underline{\underline{q}} = - \frac{\partial \psi(\underline{\underline{\epsilon}}^e, \underline{\underline{\alpha}})}{\partial \underline{\underline{\alpha}}}$$

microstructural

internal - not obs.

$\underline{\underline{q}}$ - internal vars - we cannot directly observe

Some dissipation mechanism, has to be mathematically described by some thermodynamic conjugate variables - hardening.

$\underline{\underline{q}}$ is the thermodynamic conjugate of $\underline{\underline{\alpha}}$

Strain energy → elastic tensor
hardening variables

$$\psi(\underline{\underline{\epsilon}}^e, \underline{\underline{\alpha}}) = W_e(\underline{\underline{\epsilon}}^e) + W_h(\underline{\underline{\alpha}})$$

→ From linear elasticity, for small deformation.

$$W_e = \frac{1}{2} \underline{\underline{\epsilon}}^e : \underline{\underline{C}} : \underline{\underline{\epsilon}}^e$$

Hook's law

Variational form

→ For hardening potential

$$W_h = \frac{1}{2} H \hat{\underline{\underline{\alpha}}}^2 + \frac{1}{3} H |\underline{\underline{\alpha}}|^2$$

Kinematic 2nd order scalar

Isotropic hardening

$$\underline{\underline{q}} = - \frac{2}{3} H \underline{\underline{\alpha}}, \quad q_{ij} = - \frac{2}{3} H \alpha_{ij}$$

isotropic hardening

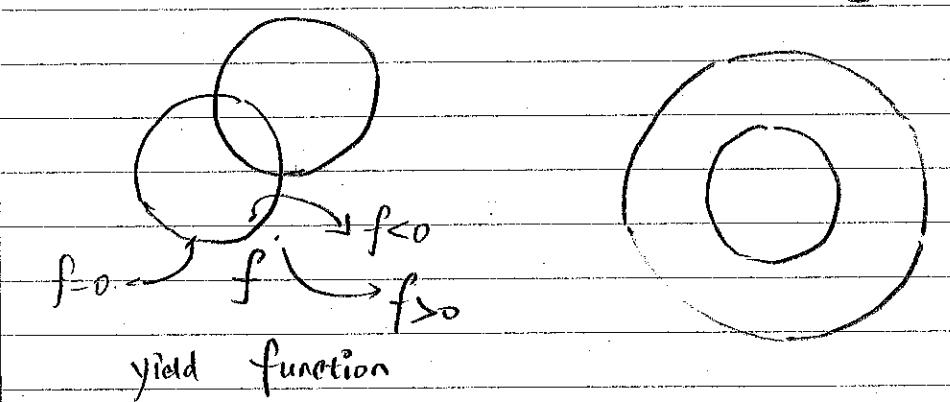
and $\hat{\underline{\underline{q}}} = - H \hat{\underline{\underline{\alpha}}}$ Kinematic hardening

$$\begin{cases} \underline{\underline{q}} = - \frac{\partial W_r}{\partial \underline{\underline{\alpha}}} \\ \dot{\underline{\underline{q}}} = - \frac{\partial W_r}{\partial \dot{\underline{\underline{q}}}} \end{cases}$$

→ simplified version, not the only form

the yield condition: $f(\underline{\underline{\sigma}}, \underline{\underline{q}}, \dot{\underline{\underline{q}}}) \leq 0$

\downarrow \downarrow
 kinematic isotropic
 hardening



in 1D: $f = |\underline{\underline{\sigma}}| - \underline{\underline{\sigma}}_Y \hat{\underline{\underline{\alpha}}}$ modulate the yield stress

$\underline{\underline{S}} = \text{dev } \underline{\underline{\sigma}}, \quad (\underline{\underline{S}} - \underline{\underline{q}}) \cdot (\underline{\underline{S}} - \underline{\underline{q}})$
 $\underline{\underline{L}} = \underline{\underline{S}} - \underline{\underline{q}}, \quad \underline{\underline{L}} = \text{tr}(\underline{\underline{S}} - \underline{\underline{q}})$

linear isotropic & kinematic hardening

pressure independent

$$f(\underline{\underline{S}}, \underline{\underline{q}}, \dot{\underline{\underline{q}}}) = \|\underline{\underline{S}} - \underline{\underline{q}}\| - \sqrt{\frac{2}{3}} (\underline{\underline{Y}}_0 - \hat{\underline{\underline{q}}}) \leq 0$$

Von Mises plasticity.

$$f(\underline{\underline{S}}, \underline{\underline{q}}, \dot{\underline{\underline{q}}}) = \sqrt{(\underline{\underline{S}} - \underline{\underline{q}}) \cdot (\underline{\underline{S}} - \underline{\underline{q}})} - k(\hat{\underline{\underline{q}}}) \leq 0$$

\downarrow
 if no internal variables
 \downarrow
 perfect plasticity

Elasto-plastic material laws — Flow rules

associate flow rule: \rightarrow plastic flow

\downarrow
induce plasticity

$$\dot{\underline{\underline{P}}} = \lambda \frac{\partial f}{\partial \underline{\underline{S}}}$$

$$\begin{cases} \underline{\underline{\sigma}}(\underline{\underline{\epsilon}}^p, \underline{\underline{\alpha}}, \dot{\underline{\underline{\alpha}}}) \\ \underline{\underline{q}} \end{cases}$$

"associate plasticity" — define the plastic strain rate

$\dot{\underline{\underline{\alpha}}} = \lambda \frac{\partial f}{\partial \underline{\underline{q}}}, \quad \dot{\underline{\underline{\alpha}}} = \lambda \frac{\partial f}{\partial \dot{\underline{\underline{q}}}}$

time is

"quasi-time"

\downarrow \downarrow \downarrow \downarrow
 kinematic isotropic hardening quasi-static exp.

flow direction:

$$\underline{n} = \frac{\partial f}{\partial \underline{S}} = \frac{\underline{S} - \underline{q}}{\|\underline{S} - \underline{q}\|}$$

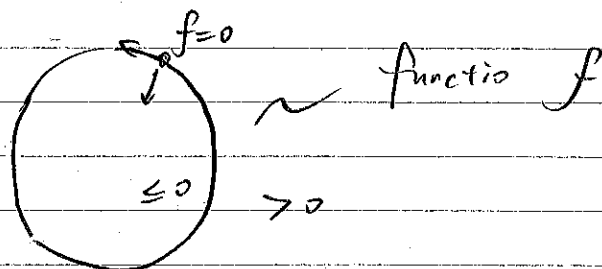
$$\frac{\partial f}{\partial \underline{q}} = -\underline{n}$$

the evolution equations:

$$\dot{\underline{e}}^p = \lambda \underline{n}, \quad \dot{\underline{\alpha}} = -\lambda \underline{n}, \quad \dot{\underline{\alpha}} = \lambda \sqrt{\frac{2}{3}}$$

the latter yields plastic strain inc.

$$\dot{\underline{\alpha}} = \sqrt{\frac{2}{3}} \|\dot{\underline{e}}^p\|, \quad \|\dot{\underline{e}}^p\| = \lambda$$



$$f = f(\underline{S}, \underline{q}, \underline{\alpha})$$

When stress is on $f=0$, distinguish 3 cases:

- { elastic unloading. $\lambda=0$. $f<0$
- { neutral loading $\lambda=0$ specifically for 3D case $f=0$
- { plastic flow. $\lambda>0$. $f=0$

KKT condition summarizes the 3 cases.

$$\lambda \geq 0, \quad f \leq 0, \quad \lambda f = 0$$

→ for any dissipative problem.

& consistency condition:

$$\dot{f} = 0, \quad \text{if } f=0$$

$$\begin{cases} f < 0, & \lambda = 0 \\ f = 0, & \lambda = 0 \\ f = 0, & \lambda > 0 \end{cases}$$

if $\lambda=0$,

derive a incremental form of small deformation plasticity; consistency for < isotropic kinematic hardening.

$$\dot{f} = \frac{\partial f}{\partial \underline{S}} \cdot \dot{\underline{S}} + \frac{\partial f}{\partial \underline{q}} \cdot \dot{\underline{q}} + \frac{\partial f}{\partial \underline{\alpha}} \dot{\underline{\alpha}} = 0$$

$$\begin{aligned} \dot{f} &= \frac{\partial f}{\partial \underline{S}} \cdot \underline{C}^e [\dot{\underline{e}} - \dot{\underline{e}}^p] + \frac{\partial f}{\partial \underline{q}} \cdot \dot{\underline{q}} + \frac{\partial f}{\partial \underline{\alpha}} \dot{\underline{\alpha}} \\ &= \underline{n} \cdot \underline{C}^e [\dot{\underline{e}}] - \lambda \left(\underline{n} \cdot \underline{C}^e [\underline{n}] + \frac{2}{3} H \underline{n} \cdot \underline{n} + \frac{2}{3} \dot{H} \right) = 0 \end{aligned}$$

the incremental rule: $\underline{\underline{\sigma}} = \text{dev} \underline{\underline{\underline{\sigma}}} + p \underline{\underline{I}}$

$$\underline{\underline{\underline{\sigma}}} = \underline{\underline{C}}^e \underline{\underline{\underline{\underline{\varepsilon}}}}$$

We can solve for plastic multiplier:

$$\lambda = A^{-1} \underline{n} \cdot \underline{\Phi^e} [\underline{\dot{e}}]$$

linear elasticity "standard" ~

Q: ① when to use specific R ?

② R_n vs. $R \dots$?

↳ \bar{v} vs. $v \dots$?