

# PERSONAL NOTES

## NONLINEAR F.E.A.

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## #FEM (Nonlinear)

▷ What is FEA?

Method to solve PDE.

▷ Why Nonlinear FEA important?

large deformation.

(linearize the strain)

↓  
plastic deformation.

(material response  
hyperelastic, etc. . . .)

▷ tools for linear FEA - relevant?

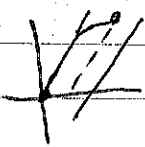
Strong form → weak form

↓  
discretize

↓  
Shape function

\* ~~At~~ local support.

Key difference:  
the solution



Prescribe the  
behavior within the  
element.

↓  
(interpolation)

↘ derivatives

↓  
solution → assembly.  
↳ postprocessing.

← Gauss Quadrature  
(integration).

# Sources of nonlinearities in solids.

• geometric nonlinearities.

• deformation

• material responses

• instabilities

• BCs.

• coupled problem.

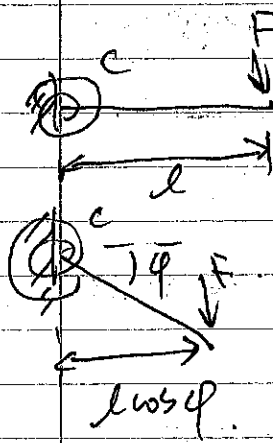
↖ force, multiphysics

- large deformation (displ.) of rigid beam

⇒ Example:

~ rigid beam, rotational spring stiffness  $c$ .

$$\sum M = 0 \rightarrow Fl \cos \varphi = c \varphi$$

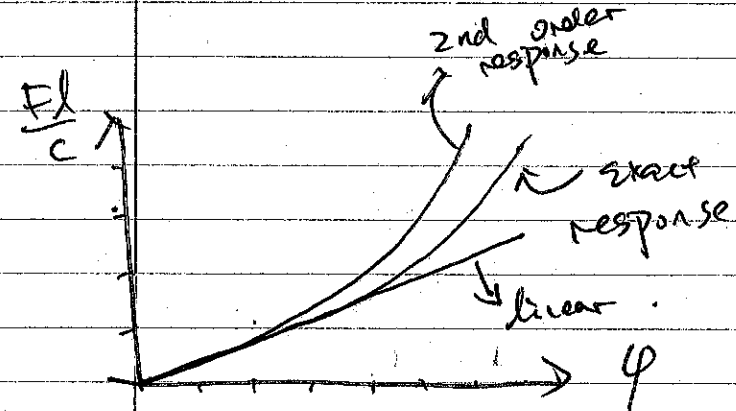


For small rotations  $\cos \varphi \rightarrow 1$ :

$$F = c \varphi / l \quad \varphi \rightarrow 0$$

to capture the nonlinearity, → 2nd order theory  
expand the function into Taylor series:

$$\cos \varphi \approx 1 - \frac{\varphi^2}{2}$$



$$F = \frac{c \varphi}{l(1 - \frac{\varphi^2}{2})}$$

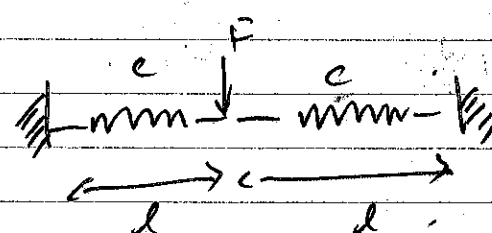
"2nd order"

\* geometric nonlinearities.

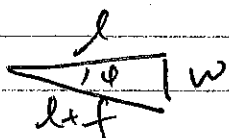
general motion.

\* this is not deformation - no strain tensor involved.

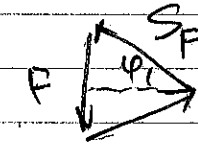
large deformation of elastic system.



$$\sum \vec{F} = 0$$



↪ elongation.



- (kinematics)

- equilibrium.

- constitutive law

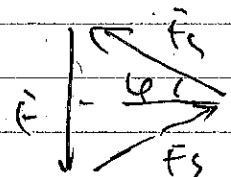
kinematics

$$w^2 + l^2 = (l+f)^2$$

$$\sin \varphi = \frac{w}{l+f}$$

equilibrium

$$2F_s \sin \varphi = F$$



constitutive law:

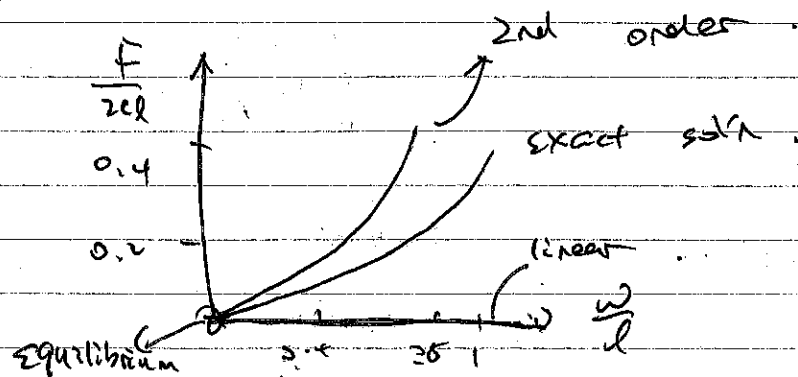
$$F_s = c f \leftarrow \text{elongation}$$

↑  
spring constant

exact sol'n:

$$\frac{w}{l} \left[ 1 - \frac{1}{\sqrt{1 + \left(\frac{w}{l}\right)^2}} \right] = \frac{F}{2cl}$$

the exact sol'n:



Taylor expansion:

$$\frac{1}{\sqrt{1 + \left(\frac{w}{l}\right)^2}} \approx 1 - \frac{1}{2} \left(\frac{w}{l}\right)^2$$

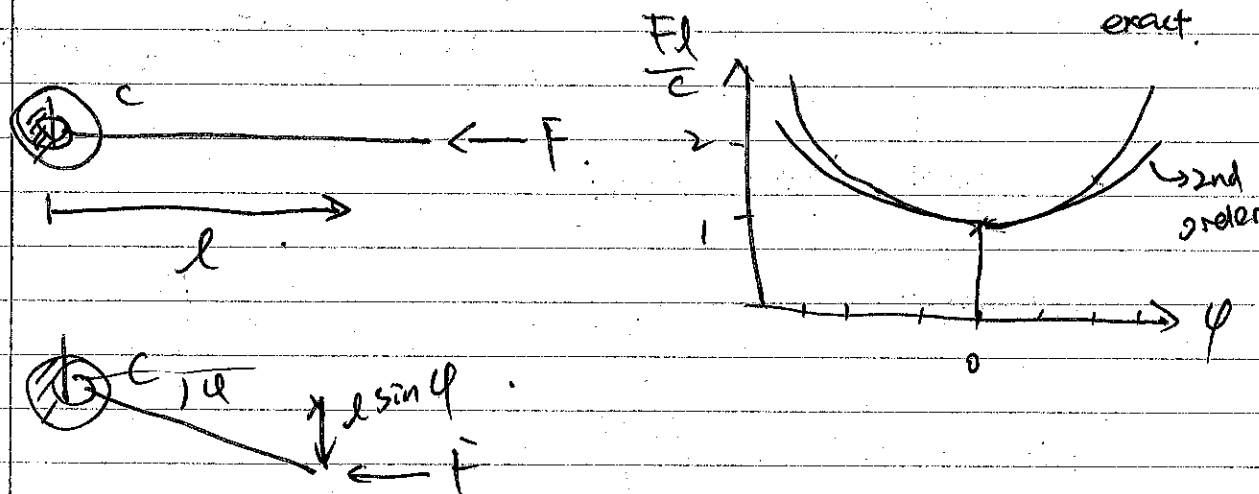
small values of  $w \rightarrow \frac{w}{l} \ll 1$

$$\frac{w}{l} \left[ \frac{1}{2} \left(\frac{w}{l}\right)^2 \right] = \frac{F}{2cl}$$

source of nonlinearities: deformation & rotation

Bifurcation

$$Fl \sin \varphi = c \varphi$$



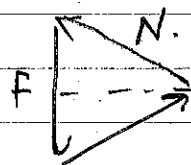
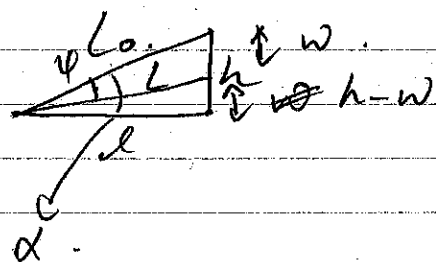
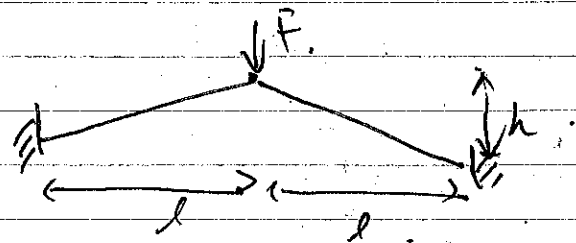
Approximation:

$$\text{Taylor expansion: } \sin \varphi \approx \varphi - \varphi^3/6$$

$$\& \quad 1/(1-x) \approx 1+x$$

Snap-through.

Geometry:  $(h-w)^2 + l^2 = L^2$  &  $L^2 + l^2 = L_0^2$



from this we can write length change:

$$f = L - L_0 = l \left[ \sqrt{1 + \left(\frac{h-w}{l}\right)^2} - \sqrt{1 + \left(\frac{h}{l}\right)^2} \right]$$

↓ final length
 ↑ initial

Equilibrium:  $N \sin(\alpha - \varphi) = -\frac{F}{2}$

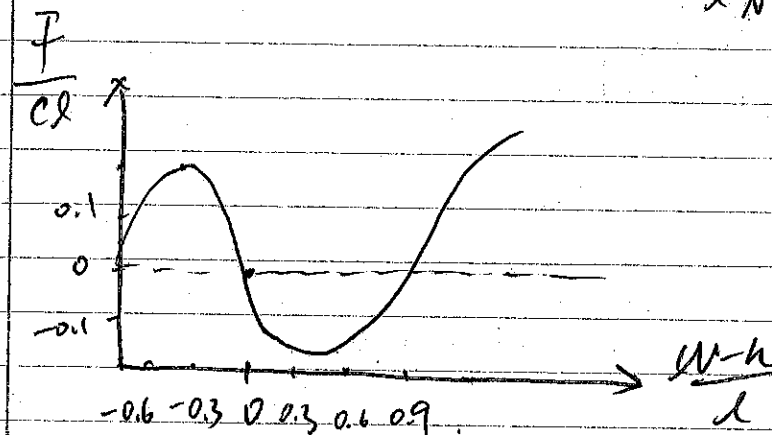
$$\sin(\alpha - \varphi) = \frac{h-w}{L}$$

$$\hookrightarrow N \frac{h-w}{L} = -\frac{F}{2}$$

Constitutive law:  $F_s = cf \rightarrow$  final expression

$$c(h-w) \frac{L-L_0}{L} = -\frac{F}{2} \rightarrow$$

$$\frac{wh}{l} \left[ 1 - \frac{L_0}{l \sqrt{1 + \left(\frac{h-w}{l}\right)^2}} \right] - \frac{F}{2cl} = 0$$



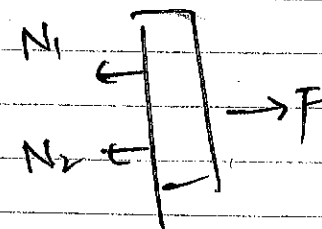
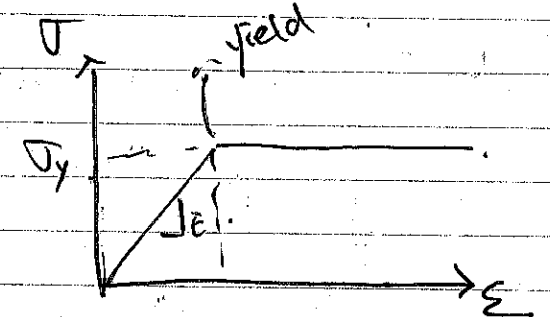
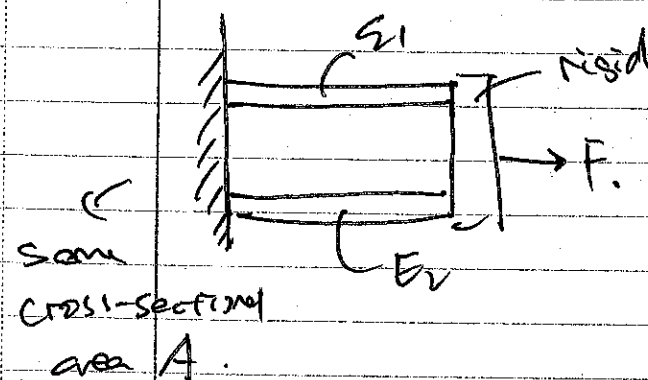
(\*)  $F$  = applied force

Material nonlinearities

elastic moduli:  $E_1 = 2E_2 = 2E$

their yield stresses:

$$\sigma_{y1} = 3\sigma_{y2} = 3\sigma_y$$



$$N_1 + N_2 = F$$

$$\sigma_1 + \sigma_2 = \frac{F}{A}$$

$$u_1 = u_2 = u, \quad \epsilon = \frac{u}{l}$$

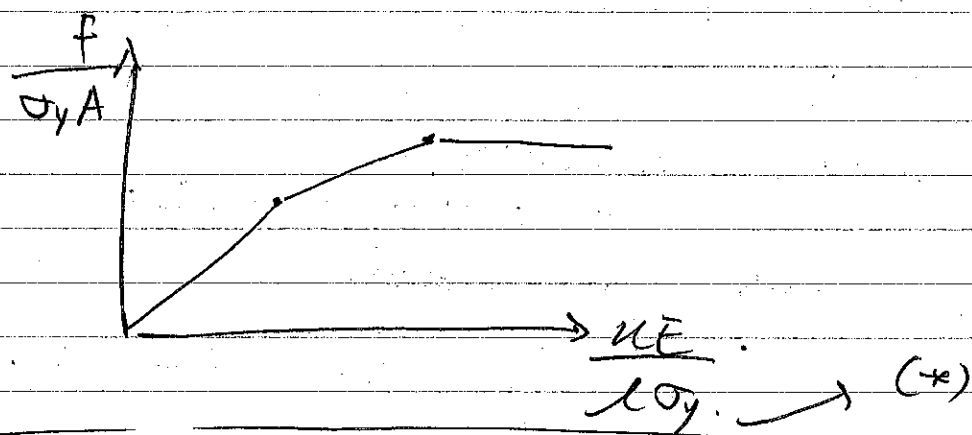
connect thru Hookes law.

$$\sigma_i = E \epsilon = E \frac{u}{l}$$

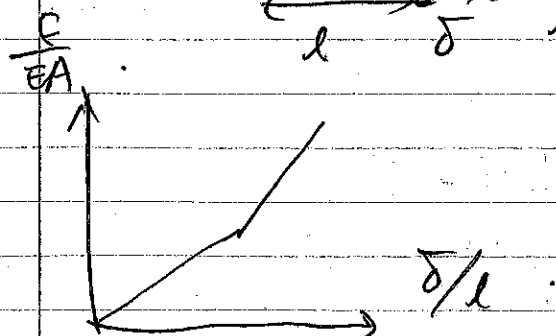
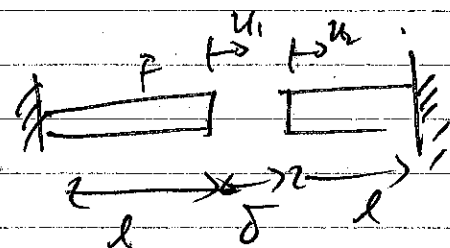
$$u = \frac{Fl}{(E_1 + E_2)A}$$

$$\sigma_y < \sigma_{y1}$$

Bar 2 yields  $\rightarrow F = 3A\sigma_y$



BC induced nonlinearity



HW: notes

$$\rightarrow \frac{\partial A^{-1}}{\partial A}$$

hint:  $A_{ik}^{-1} A_{kj} = \delta_{ij}$

$$\frac{\partial (A_{ik}^{-1} A_{kj})}{\partial A_m} = -A_{ik}^{-1} \frac{\partial A_{ik}}{\partial A_m} A_{kj}^{-1} + A_{ik}^{-1} \frac{\partial A_{kj}}{\partial A_m} A_{kj}^{-1}$$

$$\frac{\partial A^{-1} A}{\partial A}$$

$$\frac{\partial A_{ik}^{-1}}{\partial A_{kj}}$$

$$\frac{\partial A_{ik}}{\partial A_{kj}} = \delta_{ij}$$

$$\frac{\partial A_{ij}^{-1}}{\partial A_{pq}} = -A_{ip}^{-1} A_{jq}^{-1}$$

$$\frac{\partial (ab)}{\partial c} = \frac{\partial a}{\partial c} b + \frac{\partial b}{\partial c} a$$

$$\frac{\partial \det A}{\partial A} = \det(A) \cdot A^{-T}$$

$$\frac{\partial \det(A_{ij})}{\partial A_{kl}} =$$

Prob 1.  $\frac{\partial (A^{-1} A_K)}{\partial A_K}$

$$= \frac{\partial A_{ik}^{-1}}{\partial A_{pq}} \cdot A_{kj} + \frac{\partial A_{kj}}{\partial A_{pq}} \cdot A_{ik}^{-1}$$

$$\frac{\partial \delta_{ij}}{\partial A_{pq}} = \partial A_{kj} \quad \text{or} \quad = \frac{\partial \delta_{ij}}{\partial A_{pq}}$$

$$A_{ik} A_{kj}^{-1} = \delta_{ij}$$

$$\frac{\partial A_{ij}^{-1}}{\partial A_{pq}} A_{jk} + A_{ij}^{-1} \frac{\partial A_{jk}}{\partial A_{pq}} = \frac{\partial A_{ik}^{-1}}{\partial A_{pq}} A_{kk} + A_{ik}^{-1} \frac{\partial A_{kk}}{\partial A_{pq}}$$

$$\begin{aligned} \frac{\partial A_{ij}^{-1}}{\partial A_{pq}} &= -A_{il}^{-1} \frac{\partial A_{lk}}{\partial A_{pq}} A_{kj}^{-1} \\ &= -A_{il}^{-1} \delta_{lp} \delta_{kq} A_{kj}^{-1} \\ &= -A_{ip}^{-1} A_{qj}^{-1} \end{aligned}$$

Prob. 2

$$\frac{\partial \det(\underline{A})}{\partial \underline{A}} = \frac{\partial \det(\underline{A})}{\partial A_{ij}} = \frac{\partial \|\underline{A}\|}{\partial A_{ij}}$$

$$= \sum_i \text{sgn } \Pi a$$

direct notation.

$$\det A = \begin{vmatrix} a_{11} & & \\ & \dots & \\ & & \end{vmatrix}$$

$$\det \underline{A} = \|A_{ij}\|$$

$$\sum_i \sum_l \frac{\partial A_i}{\partial A_l} \cdot \frac{\partial A_{kl}}{\partial A_{ij}}$$

$$\frac{\partial \det A}{\partial A_{ij}} = \sum_k^n \sum_l^n C_{kl} \delta_{ik} \delta_{jl} = L_3 A^T$$

$$\frac{\partial \det A}{\partial A_{ij}} = \sum_k \sum_l$$

$$f(\underline{A}) = \det(\underline{A}).$$

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1 variant:

$$\left\{ \begin{aligned} \frac{\partial L}{\partial A} &= \underline{I} \\ \frac{\partial L}{\partial A} &= \underline{L}_1 \underline{I} - A^T \\ \frac{\partial L}{\partial A} &= (\underline{A}^T)^T - \underline{L}_1 A^T \end{aligned} \right.$$

$$\det(A)$$

$$A_1 A_2 - A_2 A_1$$

$$\det(\underline{A}) = \frac{1}{6} \epsilon_{ijk} \epsilon_{abc} A_{ia} A_{jb} A_{kc}$$

$$\frac{\partial}{\partial A_{ij}} \sum_k \epsilon_{ijk} \epsilon_{abc} A_{ia} A_{jb} A_{kc}$$

$$\frac{\partial}{\partial A_{ij}}$$

$$\frac{\partial \det A}{\partial A_{ij}} = (A_{ji}^T)^2 - [A_{ji}^T + I_r]$$

$$= (A_{ji}^T)^2 - \text{tr } A_{ji} A_{ji}^T$$

$$+ \frac{1}{2} [(\text{tr } A_{ji})^2 - \text{tr } A_{ji}^2]$$

$$L_3 = \det A_{ji}$$

$$L_1 = \text{tr } A_{ji} = A_{ji}^T - \text{tr } A_{ji} A_{ji}^T + \frac{1}{2} [ \sim ]$$

$$L_2 = \frac{1}{2} [(\text{tr } A)^2 - \text{tr } (A^2)]$$

$$= \det A_{ji} A_{ji}^T$$

$$= A_{ji}^T - \text{tr } A_{ji} A_{ji}^T + \frac{1}{2} [(\text{tr } A_{ji})^2 - \text{tr } A_{ji}^2]$$

$$\frac{\partial (A_{11} A_{22} - A_{12} A_{21})}{\partial} = A_{22} - A_{11}$$

Prob. 3

$$\underline{D} = \underline{a} \otimes \underline{b}$$

$$= \begin{bmatrix} a_1 b_1 & a_1 b_2 & a_1 b_3 \\ a_2 b_1 & a_2 b_2 & a_2 b_3 \\ \vdots & \vdots & \vdots \end{bmatrix}$$

$$\frac{\partial \det(A)}{\partial A} = \text{tr}(A^T \frac{\partial A}{\partial A})$$

$$\frac{\partial \det(A)}{\partial A} = \lim_{h \rightarrow 0} \frac{\det(A+h) - \det A}{h}$$

$$= \det A \lim_{h \rightarrow 0} \frac{\det[A^{-1}(A+h)] - 1}{h}$$

$$= \det$$

$$\det(\frac{\partial A}{\partial A}) = \det A$$



Prob 4.  $\underline{A} = \alpha(\underline{I} - \underline{e}_1 \otimes \underline{e}_1) + \beta(\underline{e}_1 \otimes \underline{e}_2 + \underline{e}_2 \otimes \underline{e}_1)$

compute - eigen values

- eigenvectors

we can first write out the matrix form

$$\begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} = \begin{bmatrix} \alpha & \beta \\ \beta & \alpha \end{bmatrix}$$

Prob 5. - vector form

$$\oint_C \underline{\nabla} \times \underline{F} d\underline{s} = \iint_S \text{curl } \underline{F} d\underline{S}$$

$$\iint_S \text{curl } \underline{F} d\underline{S} = \iiint_V \text{div}(\text{curl } \underline{F}) dV$$

$$\therefore \text{curl}(\text{curl } \underline{F}) = \nabla(\nabla \cdot \underline{F}) - \nabla^2 \underline{F}$$

$$\iiint_V \text{div}(\text{curl } \underline{F}) dV = \iiint_V [\nabla(\nabla \cdot \underline{F}) - \nabla^2 \underline{F}] dV$$

$$\iiint_V [\nabla(\nabla \cdot \underline{F}) - \nabla^2 \underline{F}] dV = \iint_S (\nabla \cdot \underline{F}) \underline{n} dS - \iint_S \underline{F} \cdot \underline{n} dS$$

$$\oint_C \underline{\nabla} \times \underline{F} d\underline{s} = \iint_S (\nabla \cdot \underline{F}) \underline{n} dS - \iint_S \underline{F} \cdot \underline{n} dS$$

Prob. 5

$$\oint_C \phi d\underline{x} = \int_S \underline{n} \times \text{grad } \phi dS$$

$$\oint_C \underline{u} \times d\underline{x} = \int_S [(\text{div } \underline{u}) \underline{n} - (\text{grad } \underline{u}) \cdot \underline{n}] dS$$

$$\oint_C \phi d\underline{x} = \int_S \underline{n} \times \text{grad } \phi dS$$

$$\oint_C \underline{u} \times d\underline{x}$$

$$\int_C \underline{A} \cdot d\underline{x} = \int_S \underline{A} \times d\underline{S}$$

$$\oint_C \phi d\underline{x} = \int_S \underline{n} \times \text{grad } \phi dS$$

Proof for 1:

$$\begin{aligned} \oint_C \phi d\underline{x} &= \int_C d(\underline{x} \cdot \text{grad } \phi) \\ &= \int_C d\underline{x} \times \text{grad } \phi dS \end{aligned}$$

$$= \int_S \underline{n} \times \text{grad } \phi dS$$

$d(\underline{x} \cdot \text{grad } \phi)$   
 $d\underline{x} \cdot \text{grad } \phi$   
 scalar +  $\underline{x}$

Proof for 2:

$$\int_C \underline{u} \times d\underline{x} = \int_C d(\underline{x} \times \underline{u})$$

$$= \int_S d\underline{x} \times \underline{u} dS$$

$$= \int_S [\operatorname{div} \underline{u} \underline{n} - \operatorname{grad} \underline{u} \underline{n}] dS$$

Gauss's Theorem

$$\int_S \underline{\phi} \cdot d\underline{S} = \int_V \operatorname{div} \underline{\phi} dV$$

$$\int_C \underline{\phi} d\underline{x} = \int_S \underline{n} \times \operatorname{grad} \phi dS$$

Consider efflux  $dE: dE = \underline{\phi} \cdot d\underline{S}$

$$\operatorname{div} \underline{\phi} dV = dE = \underline{\phi} \cdot d\underline{S}$$

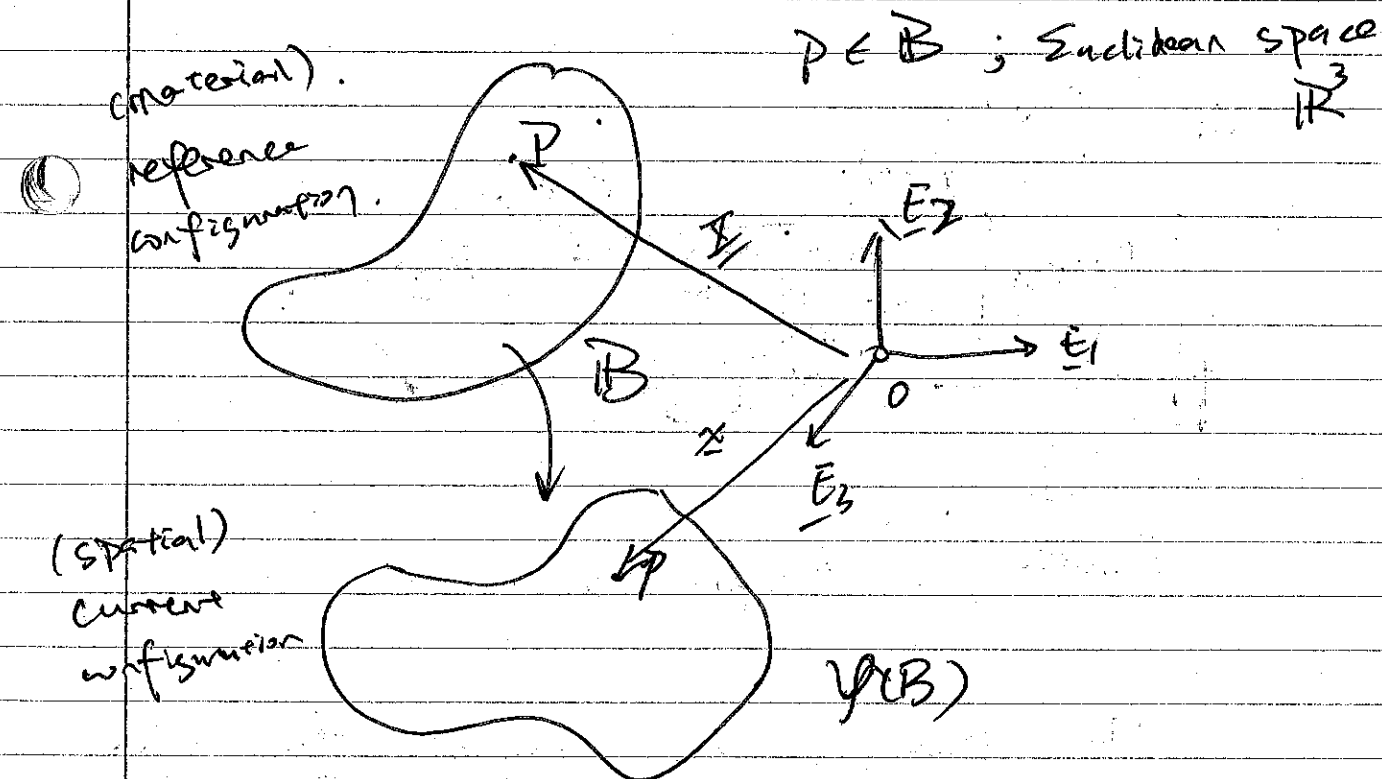
Stokes' Theorem

$$\oint_C \underline{a} \cdot d\underline{I} = \int_S \operatorname{curl} \underline{a} \cdot d\underline{S}$$

$$\sum_{\text{de loop}} \underline{a} \cdot d\underline{I} = (\nabla \times \underline{a}) \cdot d\underline{S}$$

## Lecture 2. Continuum Mechanics - important

- Kinematics (motion)
  - Strain measures.
  - vector & tensor transformation
  - Balance equations.
  - Stress measures.
  - Constitutive relations.
- this.

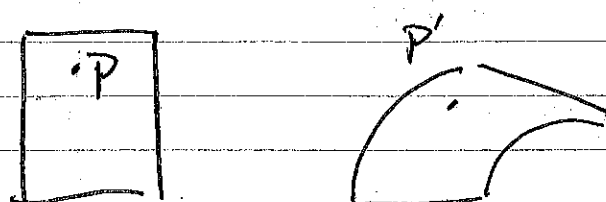


configuration  $\varphi: B \rightarrow \mathbb{R}^3$

$$\underline{x} = \varphi(\underline{X}, t)$$

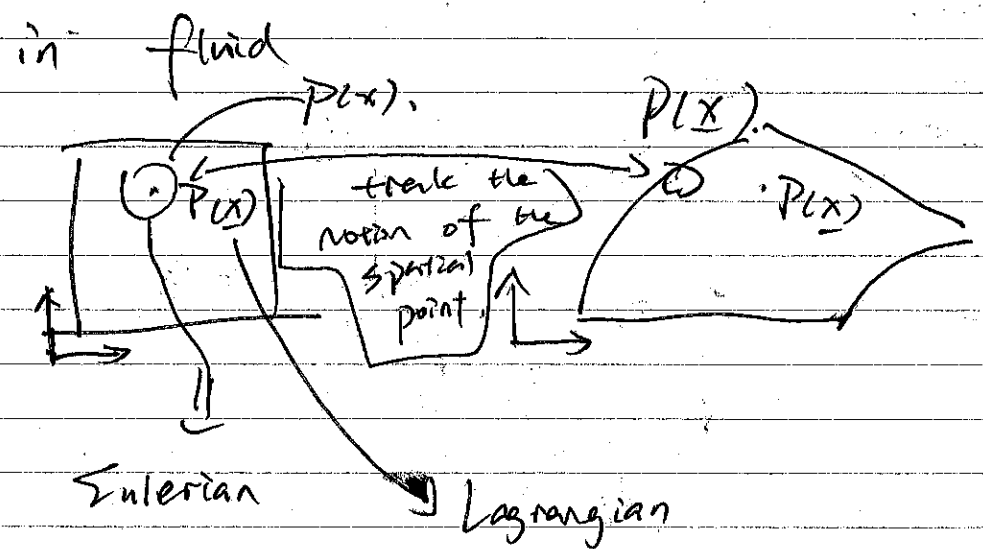
$$\underline{X} =$$

$\underline{x}, \underline{X}$  - positions in  $\mathbb{R}^3$ .  
w/ origin 0

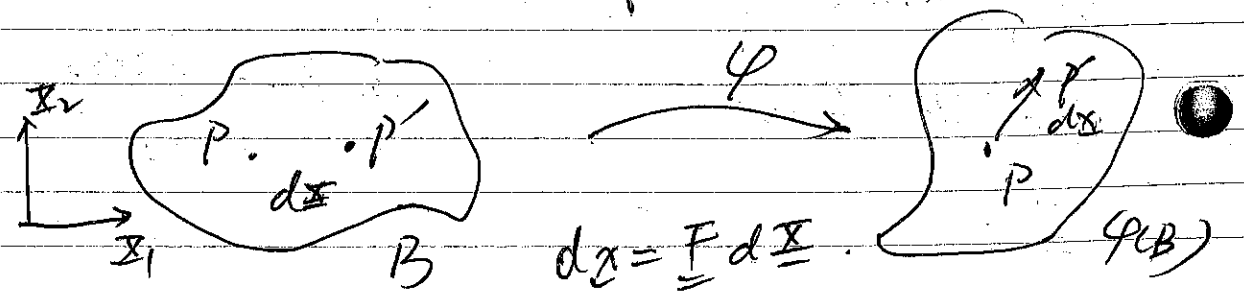


$$\underline{x} = \underline{x}_0 \underline{e}_A$$

in linear FEM - Lagrangian  
but does not make a diff.



- How line elements mapped from reference to current config.



$$\underline{F} = \frac{\partial \underline{x}}{\partial \underline{X}} \rightarrow \frac{\partial x_i}{\partial X_A} = x_{i,A}$$

$$\underline{F} = \text{Grad } \varphi(\underline{X}, t) \rightarrow \underline{x} = \varphi(\underline{X})$$

$\underline{F}$  tensor vs. matrix  
 ↓ preserves basis properties  
 ↓ numerical for n.

$$\underline{A} = \underline{a} \otimes \underline{b}$$

$$\underline{a} = a_i \underline{e}_i \quad \underline{b} = b_j \underline{e}_j$$

$$[\underline{A}] = \begin{bmatrix} \dots \end{bmatrix}$$

$$dx_i = F_{i1} dX_1 + F_{i2} dX_2 + F_{i3} dX_3$$

$$[F_{iA}] = \begin{bmatrix} \frac{\partial x_1}{\partial X_1} & \frac{\partial x_1}{\partial X_2} & \dots \\ \vdots & \vdots & \ddots \\ \frac{\partial x_3}{\partial X_1} & \dots & \dots \end{bmatrix}$$

(continued)

Kinematics - deformation gradients

$\underline{\underline{F}}$  has to be invertible

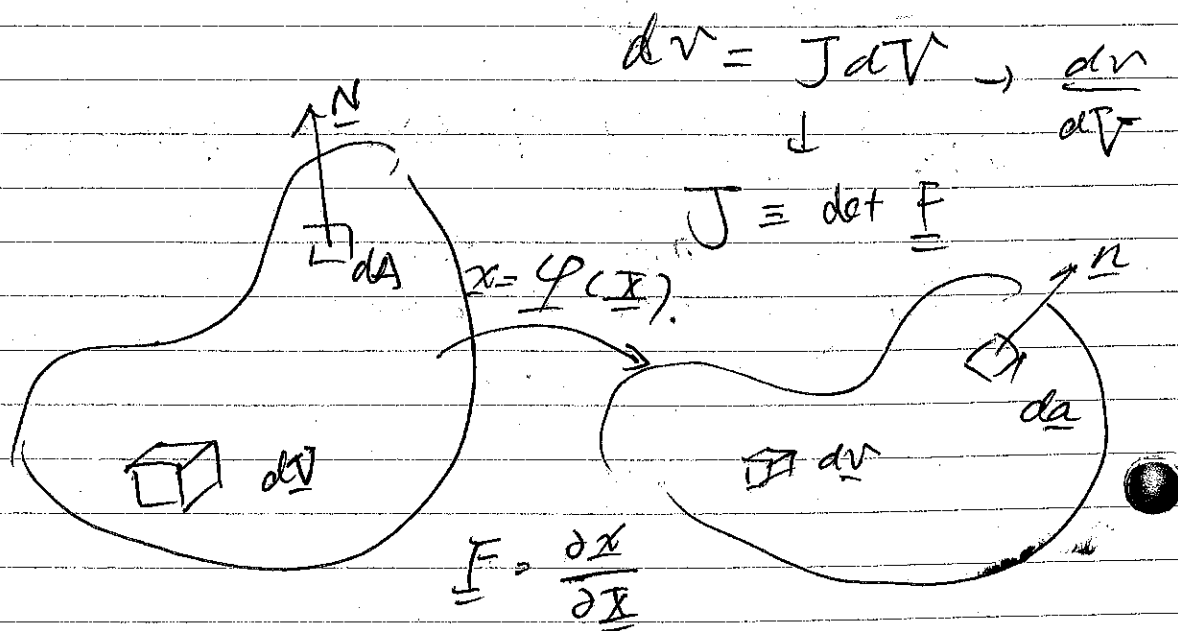
$$d\underline{\underline{X}} = \underline{\underline{F}}^{-1} d\underline{\underline{x}}$$

$$\underline{\underline{F}}^{-1} = (F^{-1})_A \underline{\underline{E}}_A \otimes \underline{\underline{e}}_i$$

$$\hookrightarrow \frac{\partial \underline{\underline{x}}_A}{\partial x_i}$$

\* Nanson's formula. Capture transformation of surface element in ref. config.  $dA$

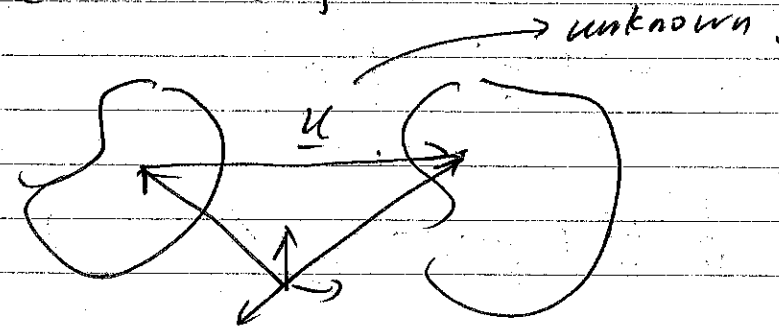
$$d\underline{\underline{a}} = \underline{\underline{n}} d\underline{\underline{a}} = J \underline{\underline{F}}^{-T} \underline{\underline{N}} dA = J \underline{\underline{F}}^{-T} d\underline{\underline{A}}$$



$$dV = J dV \rightarrow \frac{dV}{dV}$$

$$J \equiv \det \underline{\underline{F}}$$

- Define the displacement



$$\underline{\underline{u}}(\underline{\underline{X}}, t) = \underline{\underline{\phi}}(\underline{\underline{X}}, t) - \underline{\underline{X}} = \underline{\underline{x}} - \underline{\underline{X}}$$

$$\underline{\underline{F}} = \text{Grad} [\underline{\underline{X}} + \underline{\underline{u}}(\underline{\underline{X}}, t)]$$

$$= \underline{\underline{I}} + \text{Grad} \underline{\underline{u}} = \underline{\underline{I}} + \underline{\underline{H}}$$

displacement gradient

$$\text{Grad} = \nabla_{\underline{\underline{X}}}$$

- material gradient operation

$$\text{grad} = \nabla_{\underline{\underline{x}}}$$

- spatial gradient operation

$$\underline{\underline{I}} = \delta_{ij} \underline{\underline{e}}_i \otimes \underline{\underline{e}}_j$$

- Strain measures

Green - Lagrange strain tensor

Right Cauchy - Green strain tensor

$$\underline{\underline{E}} \equiv \frac{1}{2} (\underline{\underline{F}}^T \underline{\underline{F}} - \underline{\underline{I}}) = \frac{1}{2} (\underline{\underline{C}} - \underline{\underline{I}})$$

high-order terms  
get rid of

$$\underline{\underline{E}} = \frac{1}{2} (\underline{\underline{H}}^T + \underline{\underline{H}} + \underline{\underline{H}}^T \underline{\underline{H}})$$

$$\underline{\underline{\zeta}} = \frac{1}{2} (\underline{\underline{H}} + \underline{\underline{H}}^T) = \frac{1}{2} (\underline{\underline{U}})$$

Define rotation  $\underline{\underline{R}}$  ( $\underline{\underline{R}}^{-1} = \underline{\underline{R}}^T$ ). & corresponding symmetric stretch tensors

$$\underline{\underline{U}} \text{ \& \; } \underline{\underline{V}} : \underline{\underline{F}} = \underline{\underline{R}} \underline{\underline{U}} = \underline{\underline{V}} \underline{\underline{R}}$$

$$\begin{cases} F_{iB} \underline{e}_i \otimes \underline{e}_B = (R_{iA} \underline{e}_i \otimes \underline{e}_A) (U_{cB} \underline{e}_c \otimes \underline{e}_B) \\ F_{iB} \underline{e}_i \otimes \underline{e}_B = (V_{ik} \underline{e}_i \otimes \underline{e}_k) (R_{mB} \underline{e}_m \otimes \underline{e}_B) \end{cases}$$

generalized strain measure:

$$\underline{\underline{E}}^\alpha = \frac{1}{\alpha} (\underline{\underline{U}}^\alpha - \underline{\underline{I}}) \text{ \& \; } \underline{\underline{e}} = \frac{1}{\alpha} (\underline{\underline{V}}^\alpha - \underline{\underline{I}}).$$

$$\underline{\underline{E}} = \frac{1}{2} (\underline{\underline{U}}^2 - \underline{\underline{I}})$$

$$\underline{\underline{U}} = \sum_i \lambda_i \underline{N}_i \otimes \underline{N}_i ; \underline{\underline{V}} = \sum_i \lambda_i \underline{n}_i \otimes \underline{n}_i$$

$$\underline{n}_i = \underline{\underline{R}} \underline{N}_i$$

- Strain measures.

Almansi strain tensor.

$$\underline{\underline{e}} = \underline{\underline{e}}^{(-v)} = \frac{1}{2} (\underline{\underline{I}} - \underline{\underline{V}}^{-2}) = \frac{1}{2} (\underline{\underline{I}} - \underline{\underline{b}}^{-1}) = \frac{1}{2} (\underline{\underline{I}} - \underline{\underline{F}}^T \underline{\underline{F}}^{-1})$$

$$\underline{\underline{b}} \equiv \underline{\underline{F}} \underline{\underline{F}}^T = \underline{\underline{V}} \underline{\underline{R}} \underline{\underline{R}}^T \underline{\underline{V}}^T = \underline{\underline{V}}^2$$

\* pull back  
push forward

$$G(\underline{\underline{X}}) = g(\underline{\underline{x}}).$$

$$\text{Grad } G = \underline{\underline{F}}^T \text{grad } g \Rightarrow \frac{\partial G}{\partial X_A} = \frac{\partial g}{\partial x_i} \frac{\partial x_i}{\partial X_A}$$

$$\text{grad } g = \underline{\underline{F}}^{-T} \text{Grad } G.$$

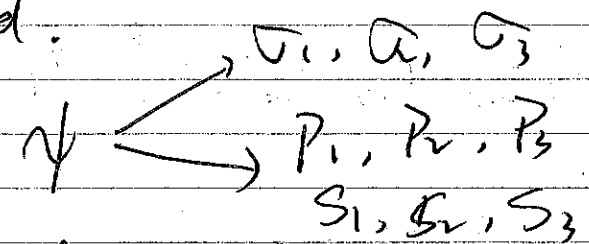
$$\frac{\partial g}{\partial x_i} = \frac{\partial g}{\partial X_A} \frac{\partial X_A}{\partial x_i}$$

pull back  
+  
push forward

Vector & tensor transformation:

HW continued.

Prob. 9.



Strain Energy function  $\Psi \rightarrow \lambda_1, \lambda_2, \lambda_3$ .

$$\frac{\partial \Psi}{\partial \lambda_i} = \sigma_i \lambda_i, \quad i=1, 2, 3.$$

$$P_i = \frac{\partial \Psi}{\partial \lambda_i} \quad \sigma_i = \lambda_i P_i.$$

$$\underline{\underline{S}} = 2\rho_0 \frac{\partial \Psi(\underline{\underline{C}})}{\partial \underline{\underline{C}}}$$

$$\underline{\underline{S}} = \underline{\underline{F}}^{-1} \underline{\underline{P}} = J \underline{\underline{F}}^{-1} \underline{\underline{Q}} \underline{\underline{F}}^{-T}.$$

$\Psi$  — incompressible isotropic hyperelastic.

i.e. o.  $\checkmark$  principal stretches.

$$\Psi = \frac{1}{2} k (\lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3)$$

$$+ \frac{k}{\beta} (\lambda_1 \lambda_2 \lambda_3)^\beta$$

$\Downarrow$   
3 Cauchy stress.

$$\begin{cases} \sigma_1 = k \lambda_1 (\lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3) + (\lambda_1 \lambda_2 \lambda_3)^\beta k \beta \lambda_1 \\ \sigma_2 = k \lambda_2 ( \quad \quad \quad ) + \quad \quad \quad \lambda_2 \\ \sigma_3 = k \lambda_3 ( \quad \quad \quad ) + \quad \quad \quad \lambda_3 \end{cases}$$

$$\Psi = (I_1, I_2, I_3) = \frac{\mu}{2} (I_1 - 3) + \frac{\lambda}{2} (I_1 - 3)^2$$

$$\Psi = \frac{\mu}{2} (I_1 - 3)$$

$$= \frac{\mu}{2} (\lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3)$$

↓

$$\sigma_1, \sigma_2, \sigma_3 = \dots$$

$$\underline{P} = -p \underline{F}^{-T} + 2 \left[ \left( \frac{\partial \Psi}{\partial I_1} + I_1 \frac{\partial \Psi}{\partial I_2} \right) \underline{F} - \frac{\partial \Psi}{\partial I_2} \underline{F} \underline{F} \right]$$

Prob 10

$$\chi_1 = \lambda_1 \underline{I}_1, \quad \chi_2 = \lambda_2 \underline{I}_2, \quad \chi_3 = \frac{1}{\lambda_1 \lambda_2} \underline{I}_3$$

★ Membrane theory:

$$\text{Ogden's theory: } \sigma = \mu (\lambda^n - 1)$$

$$W = \sum_i \frac{\alpha_i}{n_i} \left[ (\lambda_1^2 + \lambda_2^2 + \frac{1}{\lambda_1^2} + \frac{1}{\lambda_2^2})^{\frac{n_i}{2}} - 2 \right]$$

Strain energy (Wiki)  $W(\lambda_1, \lambda_2) = \sum_i \frac{\alpha_i}{n_i} (\lambda_1^{\alpha_i} + \lambda_2^{\alpha_i} + \lambda_1^{-\alpha_i} \lambda_2^{-\alpha_i} - 3)$

We know  $\lambda_1$  &  $\lambda_2$  &  $\det \underline{F} = 1$

HW note.

Prob 7: reverse to get  $\underline{X}(\underline{x})$ .

$$\begin{cases} \chi_1 = e^t \underline{I}_1 - e^{-t} \underline{I}_2 \\ \chi_2 = e^t \underline{I}_1 + e^{-t} \underline{I}_2 \\ \chi_3 = \underline{I}_3 \end{cases}$$

$$\underline{u} = \underline{x} - \underline{X}$$

$$\underline{F} = \frac{\partial \underline{x}}{\partial \underline{X}} = \frac{\partial (\underline{u} + \underline{X})}{\partial \underline{X}}$$

$$\begin{cases} \chi_1 + \chi_2 = 2e^t \underline{I}_1 \rightarrow \underline{I}_1 = \frac{1}{2e^t} (\chi_1 + \chi_2) \\ \chi_2 - \chi_1 = 2e^{-t} \underline{I}_2 \rightarrow \underline{I}_2 = \frac{1}{2e^{-t}} (\chi_2 - \chi_1) \\ \underline{I}_3 = \chi_3 \end{cases}$$

in the material description:

$$\underline{A}(\underline{X}, t) = \frac{\partial \underline{V}(\underline{X}, t)}{\partial t} = \begin{cases} \frac{\partial \chi_1}{\partial t} = e^t \underline{I}_1 - e^{-t} \underline{I}_2 \\ \frac{\partial \chi_2}{\partial t} = e^t \underline{I}_1 + e^{-t} \underline{I}_2 \\ \frac{\partial \chi_3}{\partial t} = 0 \end{cases}$$

$$\underline{V}(\underline{X}, t) = \frac{\partial \underline{x}(\underline{X}, t)}{\partial t} = \begin{cases} \frac{\partial \chi_1}{\partial t} = e^t \underline{I}_1 + e^{-t} \underline{I}_2 \\ \frac{\partial \chi_2}{\partial t} = e^t \underline{I}_1 - e^{-t} \underline{I}_2 \\ \frac{\partial \chi_3}{\partial t} = 0 \end{cases}$$

$$\frac{1}{4e^{2t}} \frac{1}{e^t} = \begin{bmatrix} e^t \underline{I}_1 + e^{-t} \underline{I}_2 \\ e^t \underline{I}_1 - e^{-t} \underline{I}_2 \\ 0 \end{bmatrix}$$

Prob. 8:

$$d\underline{u} = (\underline{F} - \underline{I}) d\underline{x}$$

$$\frac{d\underline{u}}{d\underline{x}} = \underline{F} - \underline{I}$$

$$\frac{d\underline{u}}{d\underline{x}} + \underline{I} = \underline{F}$$

$$\underline{x} - \underline{x} = \underline{u}$$

$$\frac{\partial \underline{x}}{\partial \underline{x}} - \frac{\partial \underline{x}}{\partial \underline{x}} = \frac{\partial \underline{u}}{\partial \underline{x}}$$

$$\underline{I} - \underline{F}^{-1} = \frac{\partial \underline{u}}{\partial \underline{x}}$$

$$\underline{I} - \frac{\partial \underline{u}}{\partial \underline{x}} = \underline{F}^{-1}$$

$$\underline{F}^{-1} = \begin{bmatrix} 1 & -\frac{1}{4} & 0 \\ 1 & 2 & 0 \\ 0 & 0 & -3 \end{bmatrix}$$

$$\underline{F}^{-1} = \begin{bmatrix} 0 & -\frac{1}{4} & 0 \\ 1 & -1 & 0 \\ 0 & 0 & 4 \end{bmatrix}$$

$$\underline{F} = \begin{bmatrix} 0 & -\frac{1}{4} & 0 \\ -\frac{1}{4} & -\frac{1}{4} & 0 \\ 0 & 0 & \frac{1}{4} \end{bmatrix}$$

$$\underline{F} = \frac{1}{\det(\underline{F})} \text{adj}(\underline{F})$$

$$\text{adj}(\underline{F}^{-1}) = \begin{bmatrix} 0 & \frac{1}{4} & 0 \\ 1 & 1 & 0 \\ 0 & 0 & 4 \end{bmatrix}$$

Prob. 9 & 10.

$$\frac{\partial \psi}{\partial \underline{c}} = \sum_i^3 \frac{\partial \psi}{\partial \lambda_i^2} \frac{\partial \lambda_i^2}{\partial \underline{c}}$$

$$\frac{\partial \lambda_i^2}{\partial \underline{c}} = \underline{n}_i \underline{n}_i$$

$$\hookrightarrow \frac{\partial \psi}{\partial \underline{c}} = \sum_i^3 \frac{\partial \psi}{\partial \lambda_i^2} \underline{n}_i \underline{n}_i$$

$$\underline{S} = 2 \frac{\partial \psi}{\partial \underline{c}} = \sum_i^3 \frac{2 \frac{\partial \psi}{\partial \lambda_i^2} \underline{n}_i \underline{n}_i}{\frac{\partial (\lambda_1^2 + \lambda_2^2 + \lambda_3^2 - 3)}{\partial \lambda_i^2}} \underline{n}_i \underline{n}_i$$

$$= \frac{2 \lambda_1 \underline{n}_1}{2 \cdot \underline{n}_1} + \frac{2 \lambda_2 \underline{n}_2}{2 \cdot \underline{n}_2} + \frac{2 \lambda_3 \underline{n}_3}{2 \cdot \underline{n}_3}$$

$$\underline{I} = \underline{F} \underline{S}$$

$$\underline{n}(\underline{n}_1 \underline{n}_1 + \underline{n}_2 \underline{n}_2 + \underline{n}_3 \underline{n}_3)$$

$$\hookrightarrow \underline{F} \underline{N}_i = \underline{\lambda}_i \underline{n}_i$$

$$\underline{F} \underline{N}_i = \underline{\lambda}_i \underline{n}_i$$

$$\underline{P} = \underline{F} \sum_i^3 \frac{\partial \psi}{\partial \lambda_i^2} \underline{N}_i \otimes \underline{N}_i = \sum_i^3 \frac{\partial \psi}{\partial \lambda_i^2} \underline{N}_i \otimes \underline{N}_i$$

$$\underline{U} = \underline{J}^{-1} \underline{F} \underline{P} = \frac{1}{\lambda_1 \lambda_2 \lambda_3} \underline{n}(\lambda_1 \underline{N}_1 \underline{N}_1 + \lambda_2 \underline{N}_2 \underline{N}_2 + \lambda_3 \underline{N}_3 \underline{N}_3)$$



Prob 10.

Ogden model: plane stress =  $\sigma_3 = 0$

$$\psi = \sum_i \frac{2\mu_i}{\alpha_i^2} (\bar{\lambda}_1^{\alpha_i} + \bar{\lambda}_2^{\alpha_i} + \bar{\lambda}_3^{\alpha_i} - 3) + \frac{K_1}{2} (J-1)^2$$

$$\frac{\partial \psi}{\partial \lambda_i} = \sum_i \frac{2\mu_i}{\alpha_i^2} \left(\frac{\lambda_i}{J^{1/3}}\right)^{\alpha_i}$$

const.  $\boxed{\alpha_i \lambda_i^{\alpha_i-1}}$

$$\frac{2\mu_i}{\alpha_i^2} \left(\frac{1}{J^{1/3}}\right)^{\alpha_i} \alpha_i \lambda_i^{\alpha_i-1}$$

$$= \frac{2\mu_i}{\alpha_i} \left(\frac{1}{J^{1/3}}\right)^{\alpha_i} \lambda_i^{\alpha_i-1}$$

$$\underline{F} = \frac{\partial \psi}{\partial \underline{x}} = \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \lambda_2 & 0 \\ 0 & 0 & \lambda_1 \lambda_2 \end{bmatrix}$$

$$J = 1$$

Week 3 - lecture 1.

weak form

integrability  $\uparrow$  BCs.

$u \Rightarrow u = \{u \mid u \in H^1; u = g \text{ on } \Gamma_g\}$

$w = \{w \mid w \in H^1; w = 0 \text{ on } \Gamma_g\}$

$\int_{\Omega} (\nabla w)^2 dV < \infty$   
does not blow up

\* cannot prescribe both force & displacement

"Spring"-type BCs - Robin

essential vs. natural

Dirichlet vs. Neumann.

$u = g$        $-q \cdot n_i = h$

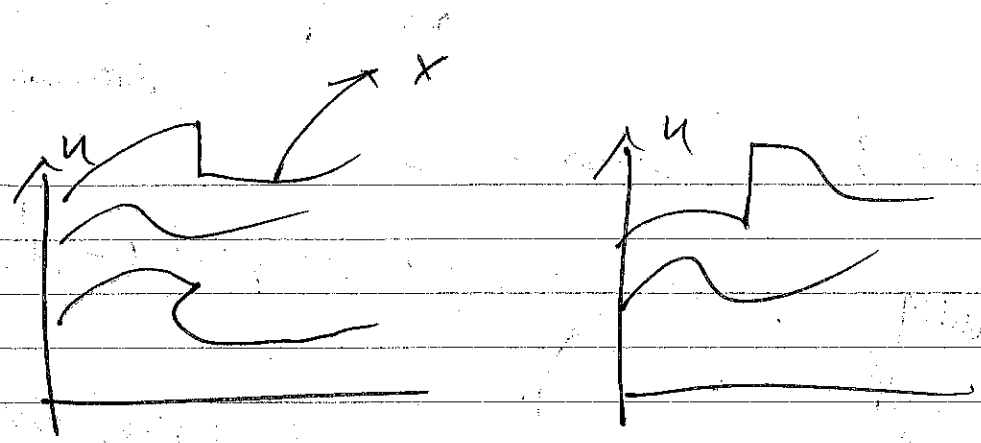
1st: multiply both sides by test function

$\Gamma = \Gamma_g \cup \Gamma_h$

$\int_{\Omega} w \operatorname{div} q = \int_{\Omega} w f$

$\int_{\Omega} w \operatorname{div} q dV = \int_{\Omega} w f dV$

$\int_{\Gamma_h} w q \cdot n dS - \int_{\Omega} q \cdot \operatorname{grad} w dV = \int_{\Omega} w f dV$



$\therefore$  we find  $u$  has to be ~~smooth~~ continuous

$C^0 \rightarrow$  zero<sup>th</sup> derivative is continuous

$C^1 \rightarrow$  first derivative is continuous

$C^n \rightarrow$   $n^{\text{th}}$  derivative is continuous.

1)  $\star$  Strong form: point-wise  
weak form: or an average sense.

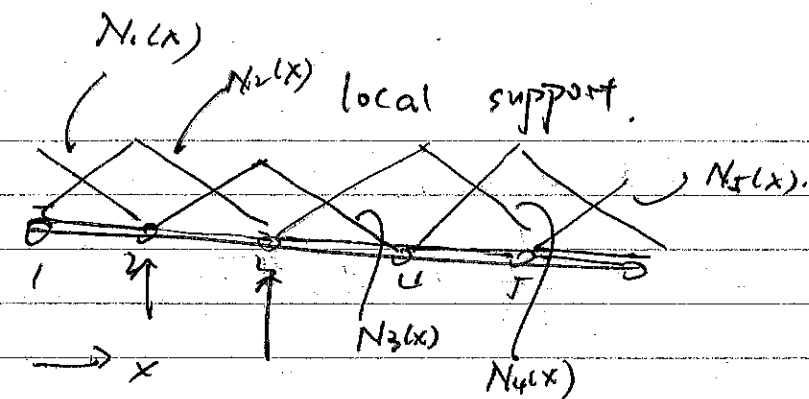
Order  
2) Strong form:  $\mathcal{Q}_{ij} = k_{ij} u_{,ij}$  - 2<sup>nd</sup> order

weak form: - 1<sup>st</sup> order deriv.

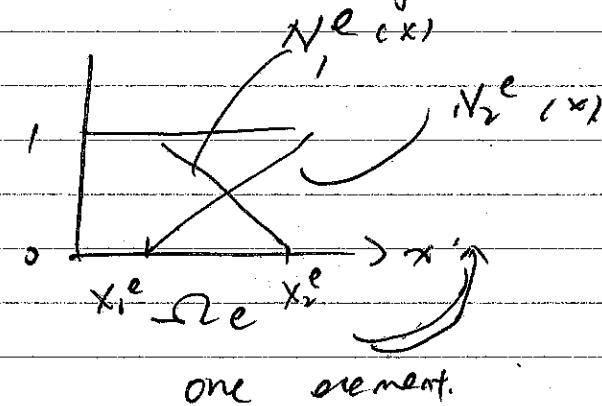
$\hookrightarrow$  symmetry between  $w$  &  $u$

- Shape function example:

1D domain:

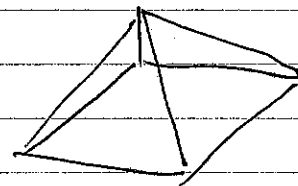
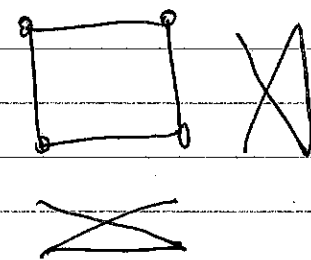
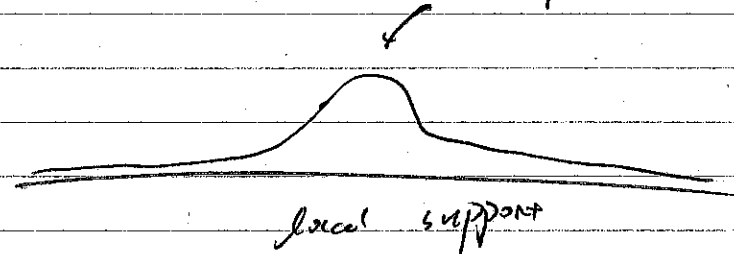


all the shape functions look the same.



- linear shape functions.

"bubble" function



$$\Rightarrow (W^h, u)_p = \sum_i C_A(N_A, u),$$

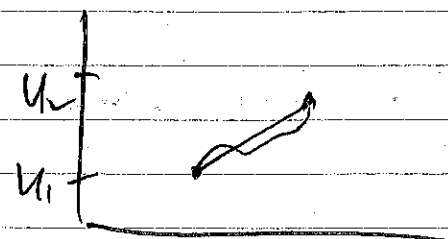
many integrals.

$$= \int_{\Gamma} N_1 C_1 ds + \int_{\Gamma} N_2 C_2 ds + \dots$$

$$\sum_{A \in \eta - \eta_b} C_A \Bigg\} \begin{matrix} \text{---} \text{---} \text{---} \end{matrix} \Bigg\} = 0 \quad \text{we can set}$$

$$u \approx u_1 N_1 + u_2 N_2$$

$$\frac{\partial u}{\partial x} \approx u_1 \frac{\partial N_1}{\partial x} + u_2 \frac{\partial N_2}{\partial x}$$

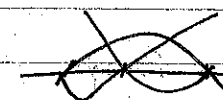


#FEM

Week 3 - Lecture 2.

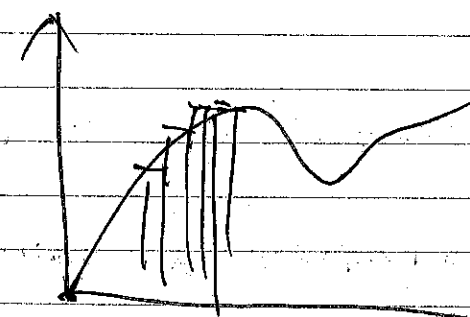
Galerkin formulation.

$$[K] \begin{Bmatrix} d_1 \\ d_2 \\ \vdots \\ d_n \end{Bmatrix} = \{F\} \rightarrow \text{solve for } d$$



quadratic

quadratic



Gauss quadrature

sample  $\rightarrow$  integration method

$\nearrow$  Same shape function  
 Bubbiu - Galerkin  $\rightarrow$  same s.f.  
 Petrov - Galerkin  $\rightarrow$  different s.f.  
 $\hookrightarrow$  subg.

how to solve PDE  $\rightarrow$  tune the shape functions

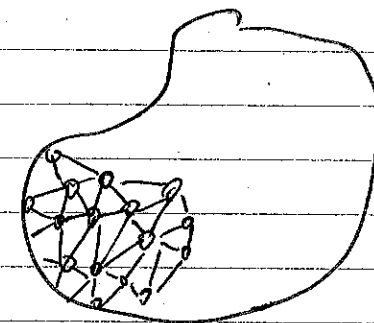
Galerkin method vs. Rayleigh Ritz method

$\Downarrow$   
 Shape function  
has to be global

$\swarrow$  Co : derivatives have a jump.

- ① what order derivatives in weak form
- ② what order derivatives in required.

Week 4. Lecture 1.



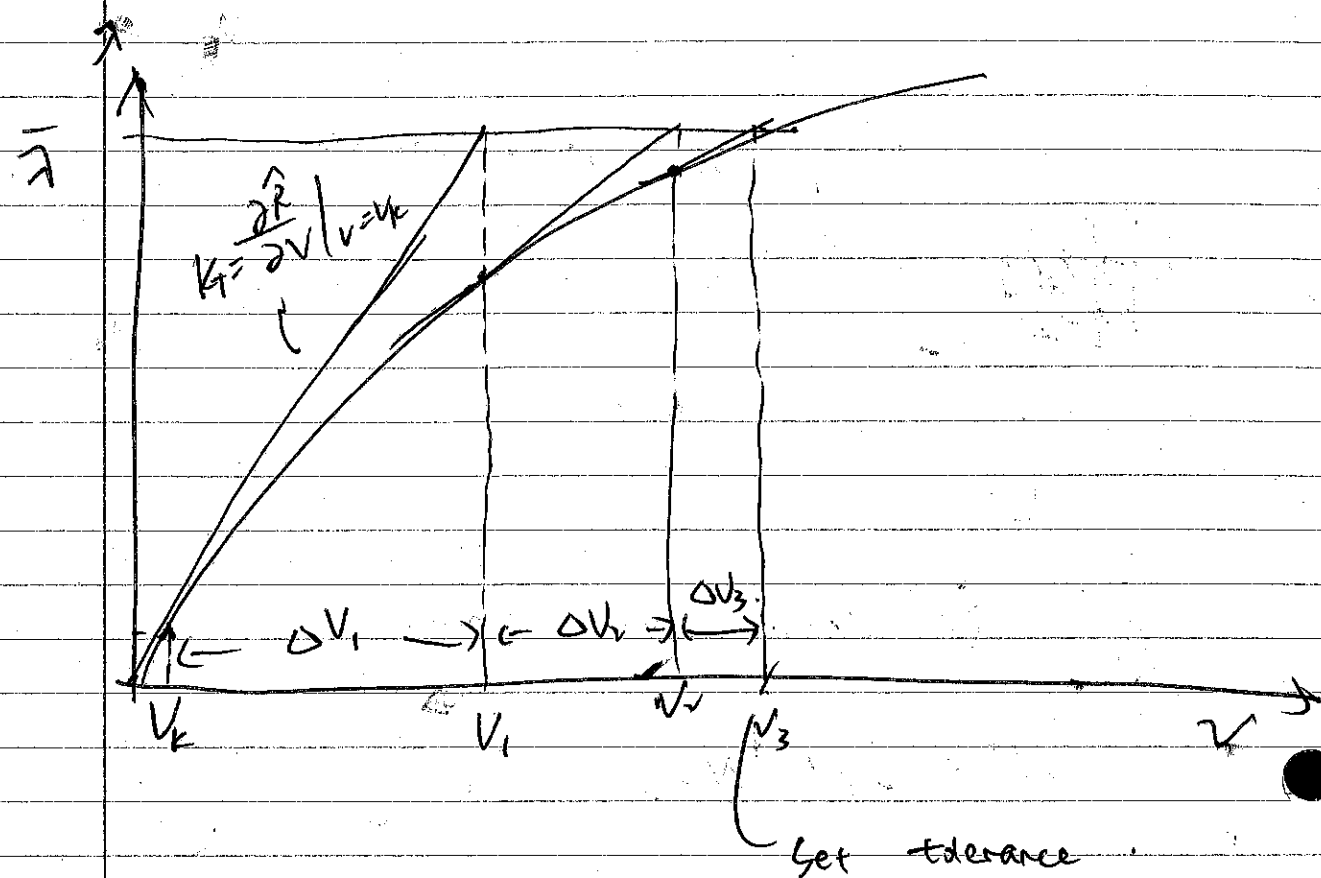
$u, u^h \rightarrow$  ele. size

$\hookrightarrow \underline{L^2}$

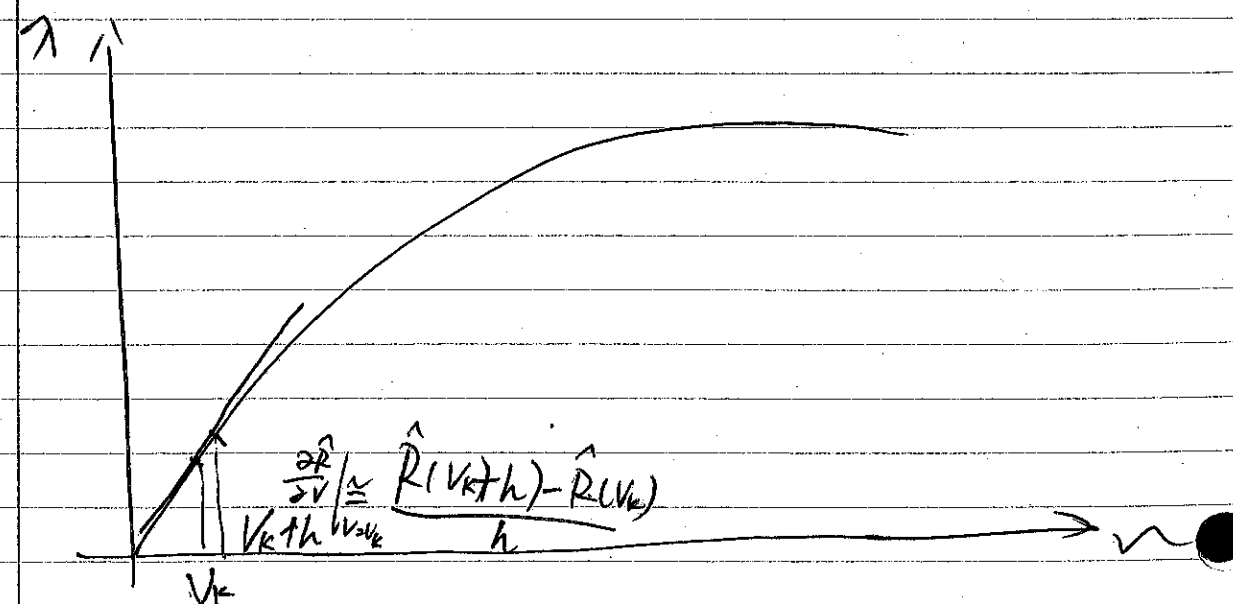
eg. 1D linear shape func.

$$\underline{N^e} = [N_1, N_2] \quad \underline{L^e} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

Week 5, Lecture 1

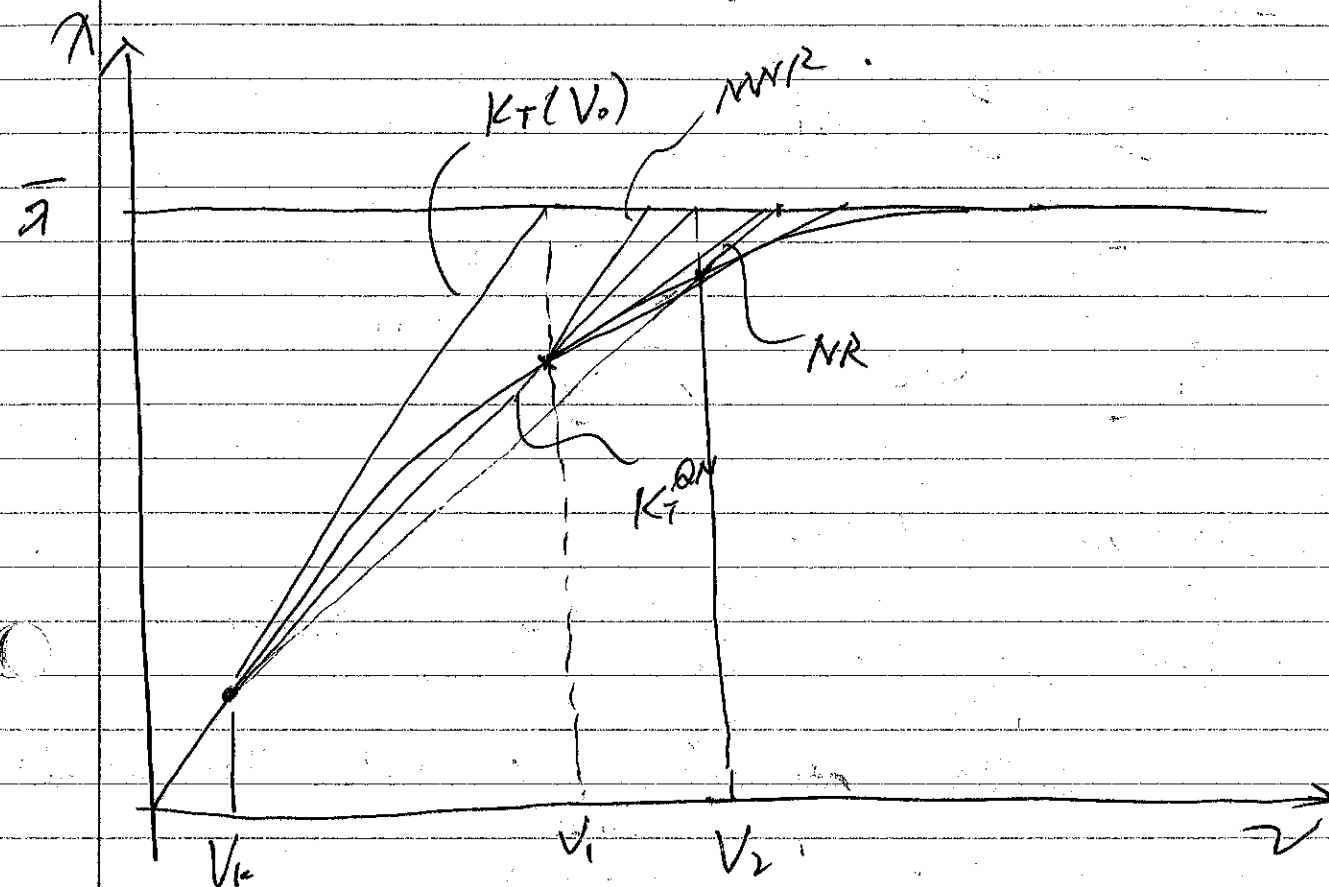


"Quadratic Convergence"



Lecture 2

Comparison between 3 methods



HW2 2

Newton - Raphson

$$\underline{G} = \underline{R} - \lambda \underline{P}$$

$$\begin{bmatrix} G_1 \\ G_2 \end{bmatrix} = \begin{bmatrix} R_1 \\ R_2 \end{bmatrix} - \lambda \begin{bmatrix} P_1 \\ P_2 \end{bmatrix} \rightarrow 0$$

$$V_0 = V_k \quad (ICs)$$

For  $\lambda_i$  in  $\lambda$  ← initialize

while  $\Sigma > 1e-4$  and iter  $\leq 15$ .

$$\underline{G} + \underline{D} \underline{G} \Delta \underline{V}$$

$$\begin{bmatrix} G_1 \\ G_2 \end{bmatrix} + \begin{bmatrix} D_1 G_1 & D_1 G_2 \\ D_2 G_1 & D_2 G_2 \end{bmatrix} \begin{bmatrix} \Delta V_1 \\ \Delta V_2 \end{bmatrix}$$

equiv.

$$\underline{K}_T \Delta \underline{V} = -\underline{G} \quad \dots (1)$$

$$\underline{V} = \underline{V} + \Delta \underline{V} \quad \dots (2)$$

$$\|\underline{G}\| = \sqrt{G[0]^2 + G[1]^2} \quad (3)$$

for  $j = [0, 1, 2, \dots, 40]$

$$\Delta \lambda = 0.25$$

$$\lambda = \Delta \lambda * j$$

$$\underline{G}_0 = \underline{K}_T \Delta \underline{V}_0$$

while  $\Sigma > 1e-4$  & iter  $\leq 15$ .

$$\underline{G} = \underline{R} - \lambda \underline{P}$$

$$\underline{K}_T = [ \quad ]$$

$$\underline{V} = \underline{V} + \Delta \underline{V}$$

$$\|\underline{G}\| = \sqrt{\dots}$$

$$\underline{G} = - \begin{bmatrix} 0.6V_1 \Delta V_1 + 6\Delta V_1 - 2xV_2 \Delta V_2 \\ -\Delta V_1 + \Delta V_2 \end{bmatrix}$$

$$\underline{K}_T = \begin{bmatrix} 0.6V_1 + 6 & -2xV_2 \\ -2xV_2 & 1 \end{bmatrix} \begin{bmatrix} \Delta V_1 \\ \Delta V_2 \end{bmatrix}$$

# Newton-Raphson implementation.

# Initialization

$$\text{for } i \text{ while } \underline{V}_0 = \underline{V}_k = \begin{bmatrix} \cdot \\ \cdot \end{bmatrix}$$

$i \leq 15$

# Compute  $\underline{G}$

$$\underline{G} = \underline{R} - \lambda \underline{P} \Rightarrow \begin{bmatrix} G_1 \\ G_2 \end{bmatrix} = \begin{bmatrix} R_1 \\ R_2 \end{bmatrix} - \lambda \begin{bmatrix} P_1 \\ P_2 \end{bmatrix}$$

# Compute  $\underline{K}_T$

$$\underline{K}_T = D \underline{G} = \begin{bmatrix} \partial G_1 / \partial V_1 & \partial G_1 / \partial V_2 \\ \partial G_2 / \partial V_1 & \partial G_2 / \partial V_2 \end{bmatrix}$$

# Compute  $\Delta V_{i+1}$

$$\Delta V_{i+1} = \frac{\underline{G}}{\underline{K}_T} = \underline{K}_T^{-1} \underline{G}$$

# Compute new  $\underline{V}$

$$\underline{V}_{i+1} = \underline{V}_i + \Delta V_{i+1}$$

$i = i + 1$

# Compute convergence

$$\epsilon = \sqrt{(\underline{G}[0])^2 + (\underline{G}[1])^2}$$

# How to solve I.C.S?

$$\begin{bmatrix} 0.2 V_1^3 - \lambda V_2^2 + 6 V_1 \\ V_2 - V_1 \end{bmatrix} = 0$$

$$0.2 V_1^3 - \lambda V_2^2 + 6 V_1$$

# HW Line Search Method

$\downarrow$

$$g(\alpha_i) = \Delta V_{i+1}^T \left[ \underline{G}(\underline{V}_i + \alpha_i \Delta V_{i+1}, \lambda) \right] = 0$$

$$\underline{G} = \begin{bmatrix} 0.2 V_1^3 - \lambda V_2^2 + 6 V_1 \\ V_2 - V_1 \end{bmatrix} - \lambda_i \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

$$g(\alpha_i) = \begin{bmatrix} \Delta V_{i+1}[0] \\ \Delta V_{i+1}[1] \end{bmatrix} \begin{bmatrix} 0.2(V_i + \alpha_i \Delta V_i)^3 - \lambda(V_i + \alpha_i \Delta V_i)^2 + 6(V_i + \alpha_i \Delta V_i) \\ V_i + \alpha_i \Delta V_i - V_i - \alpha_i \Delta V_i \end{bmatrix} - \lambda_i \begin{bmatrix} 1 \\ 0 \end{bmatrix}$$

to solve for  $\alpha_{i+1}$  next  $\alpha$ :

$$\alpha_i^{k+1} = \alpha_i^k - g(\alpha_i^k) \begin{bmatrix} \alpha_i^k - \alpha_i^{k-1} \\ g(\alpha_i^k) - g(\alpha_i^{k-1}) \end{bmatrix}$$

## BFGS Method

# Initialization

$$V_0 = V_k$$

# Compute  $G$

... (# same)

# Compute  $K_T$

... (# same).

$$\Rightarrow K_T(0).$$

For loop ...

• compute  $G$

• compute  $H \approx H_N$

$$\underline{W}_i = \underline{V}_i - \underline{V}_{i-1}$$

$$[\ ] = [\ ] - [\ ]$$

$$\underline{g}_i = \underline{G}_i - \underline{G}_{i-1}$$

$$[\ ] = [\ ] - [\ ]$$

$$\underline{a} = \frac{1}{\text{np.transpose}(\underline{g}_i) \cdot \underline{W}_i} \cdot \underline{W}_i$$

$$\underline{b}_i = - \left\{ \underline{g}_i - \left[ \frac{-\underline{W}_i^T \underline{g}_i}{\underline{W}_i^T \underline{G}_{i-1}} \right]^{\frac{1}{2}} \underline{G}_{i-1} \right\}$$

## BFGS

def: compute  $V(H, G, V)$

$$\bullet \Delta V = -HG$$

$$\bullet V = V + \Delta V$$

$$\rightarrow V_{i-1}, V_i$$

def: compute  $G(R, \lambda(P), V)$

$$G_{\text{new}} \leftarrow G = R - \lambda P$$

$$G \leftarrow G_{\text{new}} = R_{\text{new}} - \lambda P$$

$$\rightarrow G_{i-1}, G_i$$

$$G_i, G_{i-1} \leftarrow$$

$$V_i, V_{i-1} \leftarrow$$

$$\underline{W}_i = \underline{V}_i - \underline{V}_{i-1}$$

$$\underline{g}_i = \underline{G}_i - \underline{G}_{i-1}$$

$$a_i = \frac{1}{\underline{g}_i^T \underline{W}_i \underline{W}_i}$$

$$b_i = - \left\{ \underline{g}_i - \left[ \frac{-\underline{W}_i^T \underline{g}_i}{\underline{W}_i^T \underline{G}_{i-1}} \right]^{\frac{1}{2}} \underline{G}_{i-1} \right\}$$

$$\rightarrow H_i = H_v(a_i, b_i, H_{i-1})$$



for BFGS.

we have to solve exactly for  
 $V$  &  $V_{\text{prev}}$  and  $G$  &  $G_{\text{prev}}$   
before the  $\nabla$  loop:

$V_{\text{new}}$  exact solution:

def compute  $G_i(\lambda, v)$ .

$\rightarrow G_i$

def compute  $H(V_i, V_{i-1}, G_i, G_{i-1}, \lambda)$ .

$\rightarrow H$

def solve  $v(\lambda)$ .

"Solve  $v$  analytically"

$\rightarrow v$

for  $\lambda = [0, 0.25, 0.5, \dots, 1.0]$

$V_{i-1}, V_i, G_{i-1}, G_i$

while not converge:

$H = \text{compute } H$

$\Delta V = H \cdot G_i$

$\rightarrow V_{\text{new}} = V_i + \Delta V_{\text{new}}; V_i = V_{i-1}$

Solve for next  $v$ :

$$\begin{bmatrix} 0.2V_1^3 - \lambda V_2^2 + 6V_1 \\ V_2 - V_1 \end{bmatrix} - 0.25 \begin{bmatrix} 1 \\ 1 \end{bmatrix} = 0$$

$$0.2V_1^3 - \lambda V_2^2 + 6V_1 - 0.25 = 0$$

Week 6: Lecture 1.

# Weak form & variational principles

(Lecture 11).

- finite deformations

↳ geometric nonlinearities

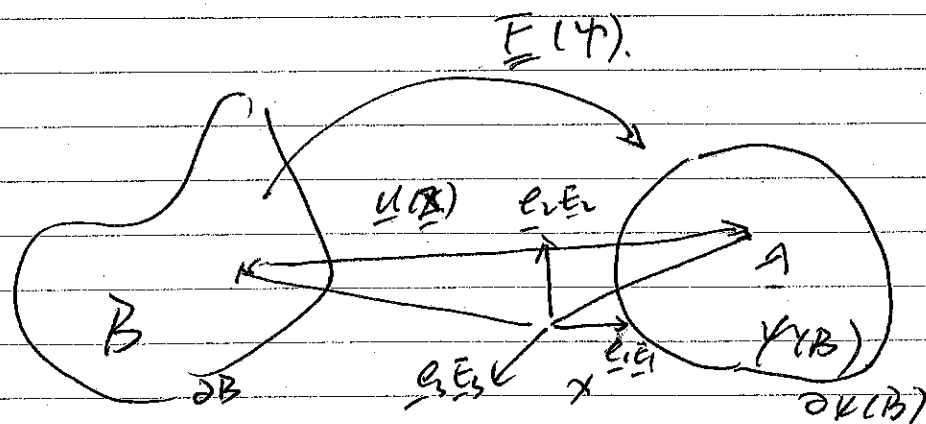
↳ material nonlinearities (const. law)

Strong form

→ Weak form

{ material description  
spatial disp.

{ material ~  
spat. ~



Strong form: hyperelasticity.

- Kinematics:

$$\underline{x} = \underline{\psi}(\underline{X}) = \underline{u} + \underline{X}$$

reference config.

$$\underline{F} = \text{Grad } \underline{\psi}(\underline{X}, t) = \underline{I} + \text{Grad } \underline{u}$$

- Constitutive law.

$$\underline{P} = \frac{\partial W(\underline{F})}{\partial \underline{F}}$$

fixed.  $\underline{\epsilon} \cdot \underline{B}$

Equilibrium:

body forces  
per unit  
ref. vol.

strain energy  
function.

$$W = W(I_1, I_2, I_3)$$

$$\text{Div } \underline{P} + \underline{b} = \rho \underline{\dot{v}} \quad \underline{\epsilon} \cdot \underline{B}$$

principle invariants  
of  $\underline{C} = \underline{F}^T \underline{F}$

Material divergence

"2nd order"

- BCs:

$$\partial B_u \cup \partial B_\sigma = \partial B \quad \underline{u} = \underline{\bar{u}} \quad \text{on } \partial B_u \quad \text{essential}$$

$$\partial B_u \cap \partial B_\sigma = \emptyset \quad \underline{P} \underline{N} = \underline{\bar{t}} \quad \text{on } \partial B_\sigma \quad \text{natural}$$

2.L. strain tensor

- kinematic  

$$\underline{\underline{E}} = \frac{1}{2} (\underline{\underline{F}}^T + \underline{\underline{F}} - \underline{\underline{I}}) \quad \underline{\underline{F}} = \underline{\underline{I}} + \text{Grad} \underline{\underline{u}}$$

- const. mod.  

$$\underline{\underline{S}} = \frac{\partial W}{\partial \underline{\underline{E}}}$$

- equilibrium  

$$\text{Div} (\underline{\underline{F}} \underline{\underline{S}}) + \underline{\underline{b}} = \rho_0 \underline{\underline{v}}$$
↑  
reference density

- B.C.s.

current config.

$\underline{\underline{b}}$  - per unit current volume

$\rho$  - current density

$\underline{\underline{D}} = \frac{1}{J} \underline{\underline{I}} \leftarrow \text{Kirchhoff stress tensor}$

lecture 13.

$\eta \cdot \underline{\underline{f}} \rightarrow \text{scalar}$

$\eta_i f_i$

Single contraction  

$$\underline{\underline{A}} \cdot \underline{\underline{B}} = A_{IK} B_{KJ} \delta_{IJ} = M_{IJ}$$

$$\underline{\underline{A}} : \underline{\underline{B}} = A_{IJ} B_{IJ} = \alpha$$

↓ trace of matrix

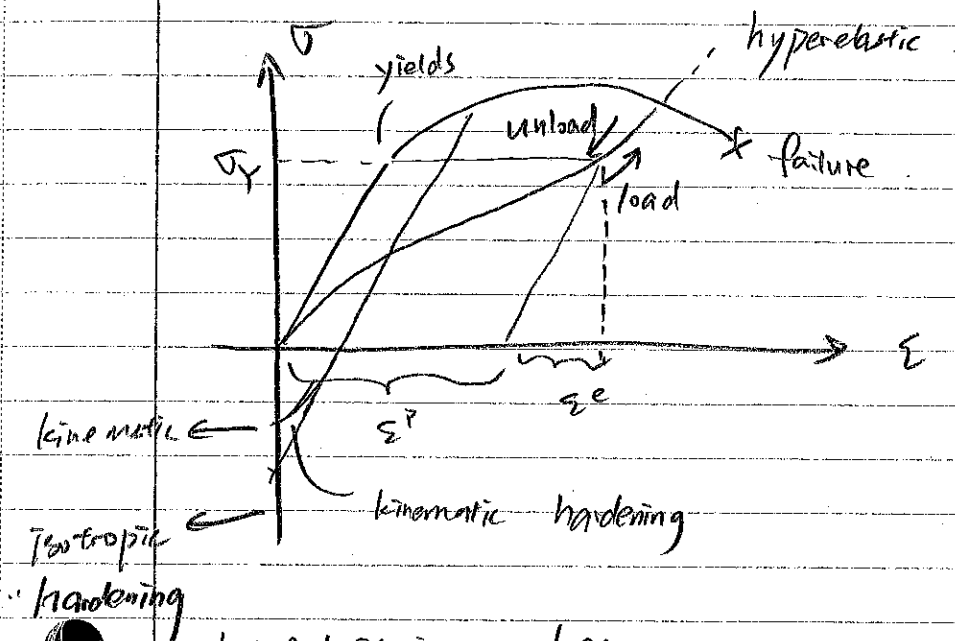
$$\underline{\underline{A}} \underline{\underline{B}} = \underline{\underline{M}}$$

$$\underline{\underline{A}} \cdot \underline{\underline{B}} = \alpha$$

Conclude: Course Notes for FEM

## Additional Notes on Plasticity

plasticity vs. hyper-elasticity.



hyperelasticity has a one-on-one mapping between the stress and strains

$$\text{Hyperelastic: } \underline{\underline{S}} = \gamma \frac{\partial W(\underline{\underline{C}})}{\partial \underline{\underline{C}}} = \gamma(\underline{\underline{C}})$$

FEA solves.

$$u = \bar{u} + \Delta u$$

↓  
C

$$G(\bar{\varphi}, \gamma) + \Delta G \cdot \Delta u \approx 0$$

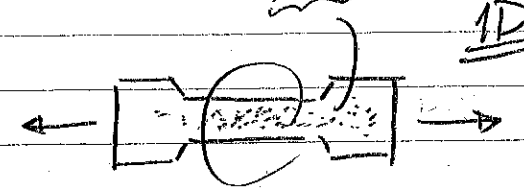
↑  
Calculate stresses in this term.

plasticity - easiest version

↓  
irreversibility.

dissipate energy to induce plastic deformation.

thinking an experiment  
 assume macroscopic pos def



homogeneous stress state

why we like it

Assume small deformations:  
 or kinematic  
 "empirical assumption from obs."

$$\underline{\underline{\epsilon}} = \underline{\underline{\epsilon}}^e + \underline{\underline{\epsilon}}^p$$

elastic      plastic

(Additive strain decomposition)

deviatoric

$$\underline{\underline{e}} = \underline{\underline{\epsilon}} - \frac{1}{3} \text{tr} \underline{\underline{\epsilon}} \underline{\underline{I}}$$

$$\underline{\underline{S}} = \underline{\underline{\sigma}} - \frac{1}{3} \text{tr} \underline{\underline{\sigma}} \underline{\underline{I}}$$

conserv. angular momentum

the same way we define a strain energy

$$\underline{\underline{\sigma}} = \frac{\partial \psi(\underline{\underline{\epsilon}}^e, \underline{\underline{\alpha}})}{\partial \underline{\underline{\epsilon}}^e}$$

stress & strain → external

$$\underline{\underline{q}} = - \frac{\partial \psi(\underline{\underline{\epsilon}}^e, \underline{\underline{\alpha}})}{\partial \underline{\underline{\alpha}}}$$

microstructural  
 internal - not obs.

$\underline{\underline{q}}$  = internal vars - we cannot directly observe

Some dissipation mechanism, has to be mathematically described by some thermodynamic conjugate variables - hardening.

$\underline{\underline{q}}$  is the thermodynamic conjugate of  $\underline{\underline{\alpha}}$

Strain energy → elastic tensor  
 → hardening variables

$$\psi(\underline{\underline{\epsilon}}^e, \underline{\underline{\alpha}}) = W_e(\underline{\underline{\epsilon}}^e) + W_h(\underline{\underline{\alpha}})$$

→ From linear elasticity, for small deformation.

$$W_e = \frac{1}{2} \underline{\underline{\epsilon}}^e : \underline{\underline{C}} : \underline{\underline{\epsilon}}^e$$

Hook's law

$\Delta \underline{\underline{E}} = \underline{\underline{C}}[\Delta \underline{\underline{\epsilon}}]$   
 Variational form

→ For hardening potential

$$W_h = \frac{1}{2} H \hat{\alpha}^2 + \frac{1}{3} H |\underline{\underline{\alpha}}|^2$$

kinematic  
 2nd order scalar

isotropic hardening

$$\underline{\underline{q}} = - \frac{2}{3} H \underline{\underline{\alpha}}, \quad q_{ij} = - \frac{2}{3} H \alpha_{ij}$$

isotropic  
 kinematic  
 hardening

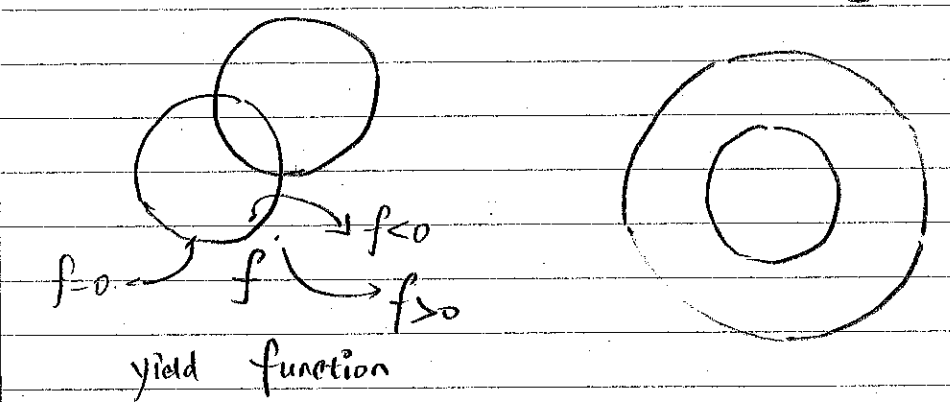
and  $\hat{q} = - H \hat{\alpha}$

$$\begin{cases} \underline{\underline{q}} = - \frac{\partial W_r}{\partial \underline{\underline{\alpha}}} \\ \dot{\underline{\underline{q}}} = - \frac{\partial W_r}{\partial \dot{\underline{\underline{q}}}} \end{cases}$$

→ simplified version, not the only form

the yield condition:  $f(\underline{\underline{\sigma}}, \underline{\underline{q}}, \dot{\underline{\underline{q}}}) \leq 0$

$\downarrow$     $\downarrow$   
 kinematic   isotropic  
                   hardening



in 1D:  $f = |\underline{\underline{\sigma}}| - \underline{\underline{\sigma}}_Y \hat{\underline{\underline{\alpha}}}$  modulate the yield stress

$\underline{\underline{S}} = \text{dev } \underline{\underline{\sigma}}, \quad (\underline{\underline{S}} - \underline{\underline{q}}) \cdot (\underline{\underline{S}} - \underline{\underline{q}})$   
 $\underline{\underline{L}} = \underline{\underline{S}} - \underline{\underline{q}}, \quad \underline{\underline{L}} = \text{tr}(\underline{\underline{S}} - \underline{\underline{q}})$

linear isotropic & kinematic hardening

pressure independent

$$f(\underline{\underline{S}}, \underline{\underline{q}}, \dot{\underline{\underline{q}}}) = \|\underline{\underline{S}} - \underline{\underline{q}}\| - \sqrt{\frac{2}{3}} (\underline{\underline{Y}}_0 - \hat{\underline{\underline{q}}}) \leq 0$$

Von Mises plasticity.

$$f(\underline{\underline{S}}, \underline{\underline{q}}, \dot{\underline{\underline{q}}}) = \sqrt{(\underline{\underline{S}} - \underline{\underline{q}}) \cdot (\underline{\underline{S}} - \underline{\underline{q}})} - k(\dot{\underline{\underline{q}}}) \leq 0$$

$\downarrow$   
 if no internal variables  
 $\downarrow$   
 perfect plasticity

Elasto-plastic material laws — Flow rules

associate flow rule:  $\rightarrow$  plastic flow

$\downarrow$   
induce plasticity

$$\dot{\underline{\underline{P}}} = \lambda \frac{\partial f}{\partial \underline{\underline{S}}}$$

$$\begin{cases} \underline{\underline{\sigma}}(\underline{\underline{\epsilon}}^p, \underline{\underline{\alpha}}, \dot{\underline{\underline{\alpha}}}) \\ \underline{\underline{q}} \end{cases}$$

"associate plasticity" — define the plastic strain rate

$\dot{\underline{\underline{\alpha}}} = \lambda \frac{\partial f}{\partial \underline{\underline{q}}}, \quad \dot{\underline{\underline{\alpha}}} = \lambda \frac{\partial f}{\partial \dot{\underline{\underline{q}}}}$

time is

"quasi-time"

$\downarrow$     $\downarrow$     $\downarrow$   
 kinematic   isotropic   hardening   quasi-static exp.

flow direction:

$$\underline{n} = \frac{\partial f}{\partial \underline{S}} = \frac{\underline{S} - \underline{q}}{\|\underline{S} - \underline{q}\|}$$

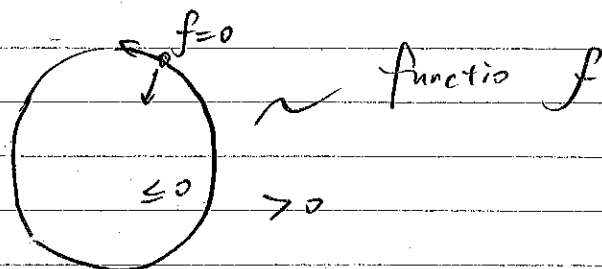
$$\frac{\partial f}{\partial \underline{q}} = -\underline{n}$$

the evolution equations:

$$\dot{\underline{e}}^p = \lambda \underline{n}, \quad \dot{\underline{\alpha}} = -\lambda \underline{n}, \quad \dot{\underline{\alpha}} = \lambda \sqrt{\frac{2}{3}}$$

the latter yields plastic strain inc.

$$\dot{\underline{\alpha}} = \sqrt{\frac{2}{3}} \|\dot{\underline{e}}^p\|, \quad \|\dot{\underline{e}}^p\| = \lambda$$



$$f = f(\underline{S}, \underline{q}, \underline{\alpha})$$

When stress is on  $f=0$ , distinguish 3 cases:

- { elastic unloading.  $\lambda=0$ .  $f<0$
- { neutral loading  $\lambda=0$  specifically for 3D case  $f=0$
- { plastic flow.  $\lambda>0$ .  $f=0$

KKT condition summarizes the 3 cases.

$$\lambda \geq 0, \quad f \leq 0, \quad \lambda f = 0$$

→ for any dissipative problem.

& consistency condition:

$$\dot{f} = 0, \quad \text{if } f=0$$

$$\begin{cases} f < 0, & \lambda = 0 \\ f = 0, & \lambda = 0 \\ f = 0, & \lambda > 0 \end{cases}$$

if  $\lambda=0$ ,

derive a incremental form of small deformation plasticity; consistency for < isotropic kinematic hardening.

$$\dot{f} = \frac{\partial f}{\partial \underline{S}} \cdot \dot{\underline{S}} + \frac{\partial f}{\partial \underline{q}} \cdot \dot{\underline{q}} + \frac{\partial f}{\partial \underline{\alpha}} \dot{\underline{\alpha}} = 0$$

$$\begin{aligned} \dot{f} &= \frac{\partial f}{\partial \underline{S}} \cdot \underline{C}^e [\dot{\underline{e}} - \dot{\underline{e}}^p] + \frac{\partial f}{\partial \underline{q}} \cdot \dot{\underline{q}} + \frac{\partial f}{\partial \underline{\alpha}} \dot{\underline{\alpha}} \\ &= \underline{n} \cdot \underline{C}^e [\dot{\underline{e}}] - \lambda \left( \underline{n} \cdot \underline{C}^e [\underline{n}] + \frac{2}{3} H \underline{n} \cdot \underline{n} + \frac{2}{3} \dot{H} \right) = 0 \end{aligned}$$

the incremental rule:  $\underline{\sigma} = \text{dev} \underline{\underline{S}} + p \underline{I}$

$$\underline{\underline{\sigma}} = \underline{\underline{C}}^e \underline{\underline{S}}$$

We can solve for plastic multiplier:

$$\lambda = A^{-1} \underline{n} \cdot \underline{\Phi^e} [\underline{\dot{e}}]$$

linear elasticity "standard" ~

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Q: ① when to use specific  $R$ ?

②  $R_n$  vs.  $R \dots$ ?

↳  $\bar{v}$  vs.  $v \dots$ ?