

# Modelica IBPSA Tutorial

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... and many contributors to the library

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# Agenda

Overview of the library

Structure

Best practices and modeling hints

Hands-on tutorial

# Modelica IBPSA Library Overview

# Primary use of Modelica IBPSA Library

- Model repository for building and district energy simulation, to be used as the core of
  - AixLib
  - BuildingSystems
  - Buildings
  - IDEAS
- License
  - All development is open-source under BSD.



In 2013, a joint effort started to avoid fragmentation, collaborate on development, implement best practices and share everything open-source and free



*Attendees of the Annex 60 planning meeting at RWTH Aachen, March 11-13, 2013*

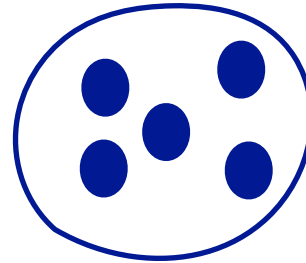


*Attendees of the first IBPSA Project 1 Expert Meeting at UdK Berlin, February 27-28, 2018*

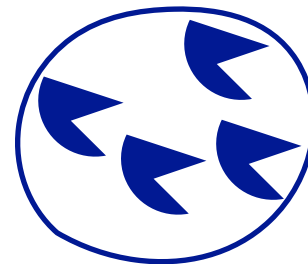


In 2013, Modelica for buildings was very fragmented. Libraries were incompatible, they replicated each other and best practices were not understood

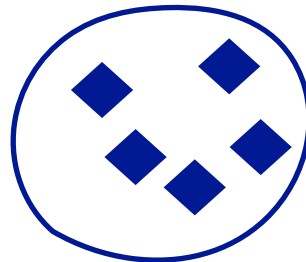
RWTH Aachen - AixLib



UdK - BuildingSystems



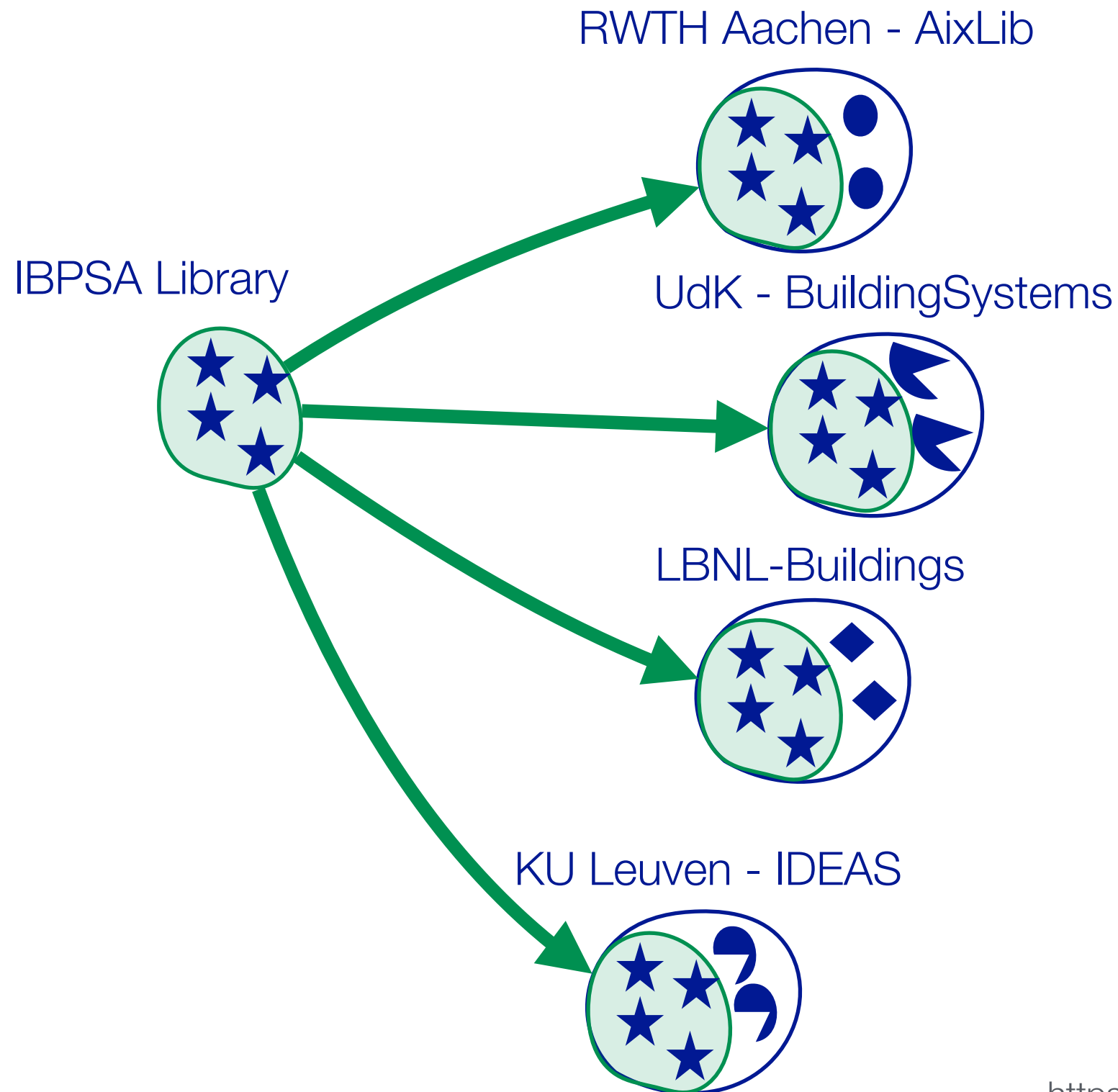
LBNL-Buildings



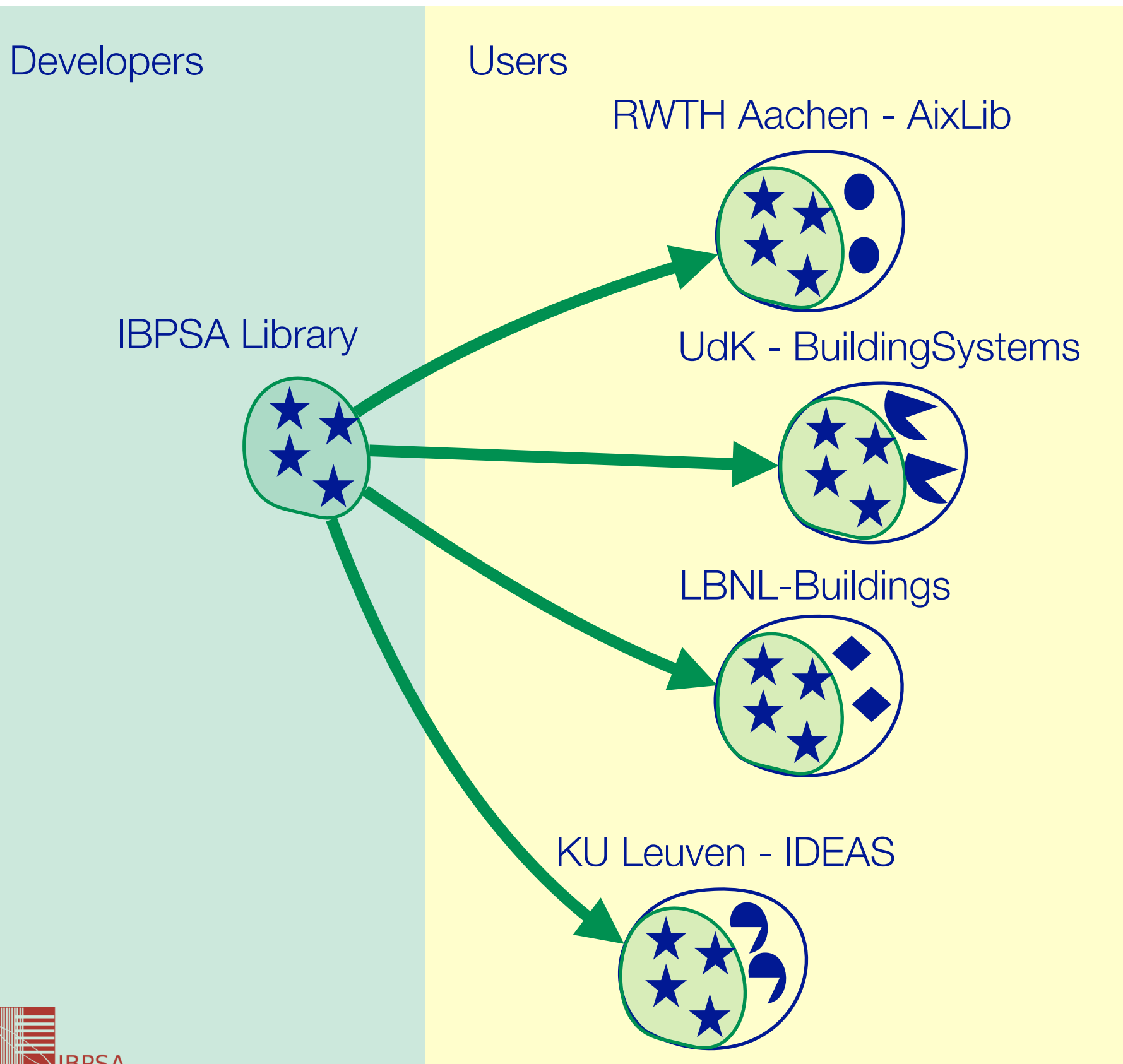
KU Leuven - IDEAS



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# Users will use derivative Modelica libraries that contain IBPSA, or tools that package these derivative libraries



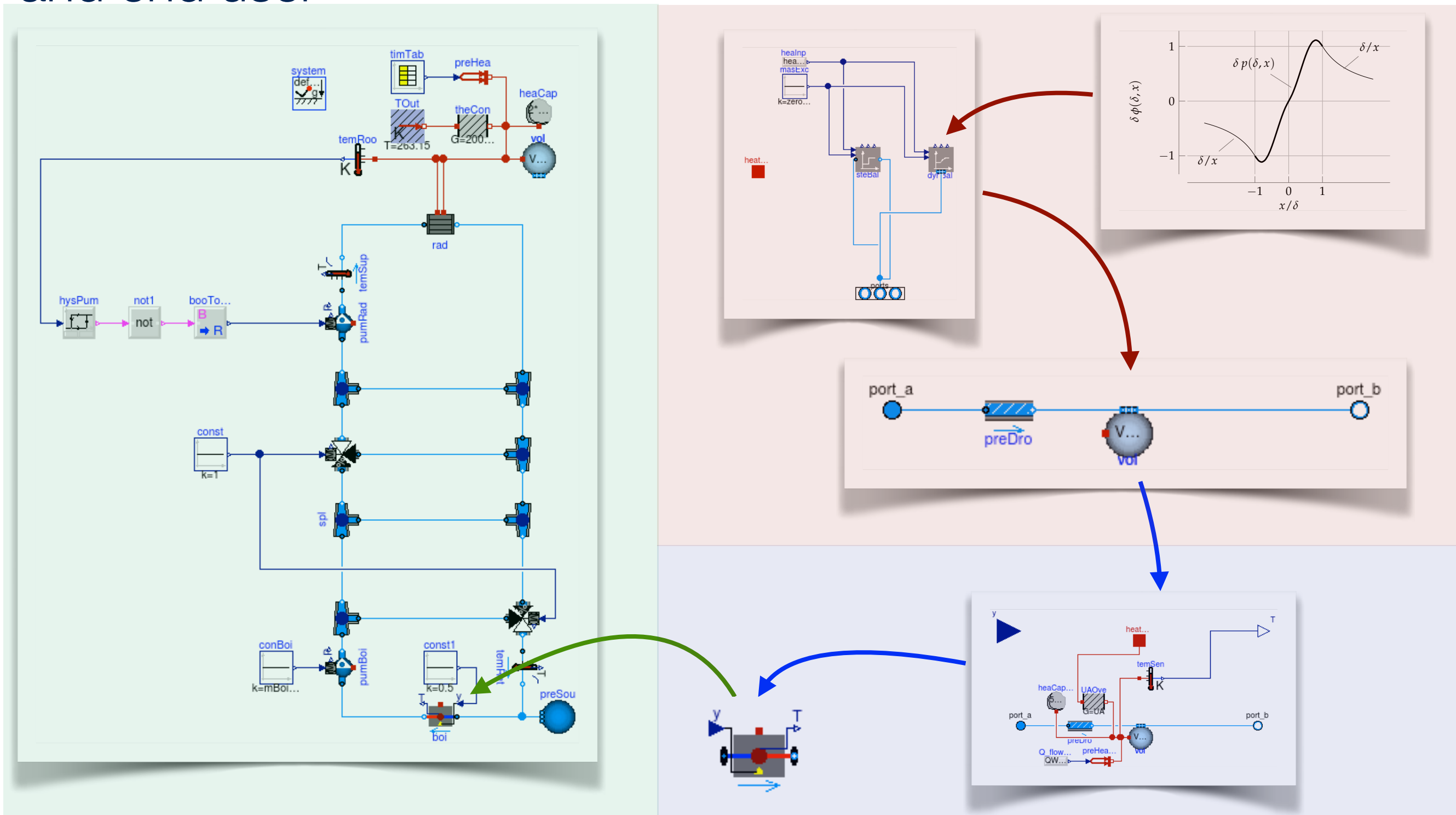
While this tutorial uses the IBPSA library, don't use it for your project work!

Instead, use any or several of the user-facing libraries.

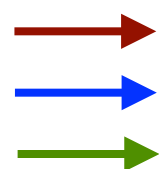
They have all functionality of the IBPSA library, plus much more.



# Separation between library developer, component developer and end user



## Legend:



Library developer

Component developer

End user

# Main modeling assumptions

## Media

Can track moisture (X) and contaminants (C).

## HVAC equipment

Most equipment based on performance curve, or based on nominal conditions and similarity laws.

Refrigerant is not modeled.

Most equipment optional steady-state or 1st order transient.

## Flow resistances

Based on  $m\_flow\_nominal$  and  $dp\_nominal$  plus similarity law.

Optional flag to linearize or to set  $dp=0$ .

## Room model

Reduced order models.

(Derivative libraries have more detailed models.)

## Electrical systems

DC.

AC 1-phase and 3-phase (dq, dq0).

Quasi-stationary or dynamic phase angle (but not frequency).

# Validation

All components are verified with analytical solutions, comparative model validation, or against guidelines such as from VDI.

600+ regression tests compare results to reference results as part of development, see <https://github.com/ibpsa/modelica-ibpsa/wiki/Unit-Tests>

# Structure of the library

# Documentation and distribution

## Documentation

- All models contain an “info” section.
- Various models contain users’ guide.
- Models in Examples and Validation packages illustrate model use.
- Derivative libraries contain additional documentation.

**INFORMATION**

Model for an air damper whose airflow is proportional to the input signal, assuming that at  $y = 1$ ,  $m\_flow = m\_flow\_nominal$ . This is unless the pressure difference  $dp$  is too low, in which case a  $k_{Dam} = m\_flow\_nominal / \sqrt{dp\_nominal}$  characteristic is used.

The model is similar to [Buildings.Fluid.Actuators.Valves.TwoWayPressureIndependent](#), except for adaptations for damper parameters. Please see that documentation for more information.

**Computation of the damper opening**

The fractional opening of the damper is computed by

- inverting the quadratic flow function to compute the flow coefficient from the flow rate and the pressure drop values (under the assumption of a turbulent flow regime);
- inverting the exponential characteristics to compute the fractional opening from the loss coefficient value (directly derived from the flow coefficient).

The quadratic interpolation used outside the exponential domain in the function [Buildings.Fluid.Actuators.BaseClasses.exponentialDamper](#) yields a local extremum. Therefore, the formal inversion of the function is not possible. A cubic spline is used instead to fit the inverse of the damper characteristics. The central domain of the characteristics having a monotonous exponential profile, its inverse can be properly approximated with three equidistant support points. However, the quadratic functions used outside of the exponential domain can have various profiles depending on the damper coefficients. Therefore, five linearly distributed support points are used on each side domain to ensure a good fit of the inverse.

Note that below a threshold value of the input control signal (fixed at 0.02), the fractional opening is forced to zero and no more related to the actual flow coefficient of the damper. This avoids stage transitions of the

## IBPSA.Fluid.Movers.UsersGuide

### Information

This package contains models for fans and pumps (movers). The same models can be used for fans or pumps.

### Model description

The models consider the pressure rise, flow rate, speed, power consumption, and heat dissipation based on the user's specification. They can take pressure rise (head), mass flow rate, or speed (absolute or relative) as control signal, and compute resulting quantities based on user-provided performance curves.

While the models in the package [IBPSA.Fluid.Movers](#) allow full customization, preconfigured models that use the same underlying physical equations are available in the package [IBPSA.Fluid.Movers.Preconfigured](#). The models in [IBPSA.Fluid.Movers](#) can also be parameterized with the data records from [IBPSA.Fluid.Movers.Data](#).

A detailed description of the fan and pump models can be found in [Wetter \(2013\)](#). The models are implemented as described in this paper, except that equation (20) is no longer used. The reason is that the transition (24) caused the derivative

$$d \Delta p(r(t), V(t)) / d r(t)$$

to have an inflection point in the regularization region  $r(t) \in (\delta/2, \delta)$ . This caused some models to not converge. To correct this, for  $r(t) < \delta$ , the term  $V(t) / r(t)$  in (16) has been modified so that (16) can be used for any value of  $r(t)$ .

Below, the models are briefly described.

### Performance data

The models use performance curves that compute pressure rise, electrical power draw and efficiency as a function of the volume flow rate and the speed. The following performance curves are implemented:

Independent variable	Dependent variable	Record for performance data	Function
Volume flow rate	Pressure	<a href="#">flowParameters</a>	<a href="#">pressure</a>
Volume flow rate	Efficiency (hydraulic or motor)	<a href="#">efficiencyParameters</a>	<a href="#">efficiency</a>
Motor part load ratio	Motor efficiency*	<a href="#">efficiencyParameters.yMot</a>	<a href="#">efficiency.yMot</a>
Volume flow rate	Power**	<a href="#">powerParameters</a>	<a href="#">power</a>

Notes (applicable to [IBPSA.Fluid.Movers.FlowControlled dp](#) and [IBPSA.Fluid.Movers.FlowControlled m flow](#)):

- \* The models will ignore this record if the nominal motor power is not provided and cannot be estimated from the pressure curve. This is because calculating the motor part load ratio requires knowing the nominal power.
- \*\* The models will ignore this record if the pressure curve is not provided and the speed is unknown. This is because the models wouldn't be able to compute the electrical power correctly using similarity laws without speed. In this case the user can mitigate the error by providing other information for hydraulic efficiency. Compare validation models [IBPSA.Fluid.Movers.Validation.PowerSimplified](#), [IBPSA.Fluid.Movers.Validation.PowerExact](#), and [IBPSA.Fluid.Movers.Validation.PowerEuler](#) as an example.

These performance curves are implemented in [IBPSA.Fluid.Movers.BaseClasses.Characteristics](#), and are used in the performance records in the package [IBPSA.Fluid.Movers.Data](#). The package [IBPSA.Fluid.Movers.Data](#) contains different data records.

### Models that use performance curves for pressure rise

The model [IBPSA.Fluid.Movers.SpeedControlled y](#) takes as an input a control signal between 0 and 1. From this input and the current flow rate, they compute the pressure rise. This pressure rise is computed using a user-provided list of operating points that defines the fan or pump curve at full speed. For other speeds, similarity laws are used to scale the performance curves, as described in [IBPSA.Fluid.Movers.BaseClasses.Characteristics.pressure](#).

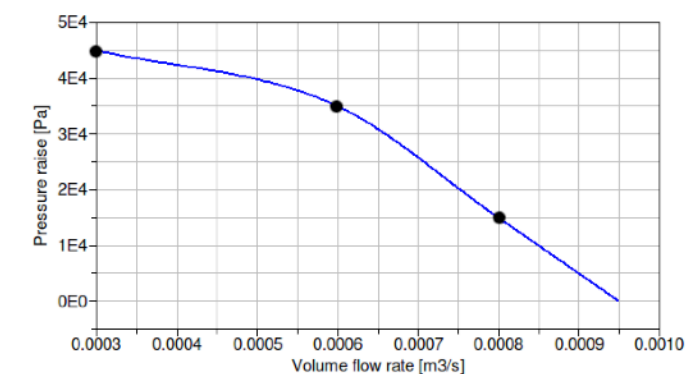
For example, suppose a pump needs to be modeled whose pressure versus flow relation crosses, at full speed, the points shown in the table below.

Volume flow rate [m³/s]	Head [Pa]
0.0003	45000
0.0006	35000
0.0008	15000

Then, a declaration would be

```
IBPSA.Fluid.Movers.SpeedControlled y pum(
  redeclare package Medium = Medium,
  per.pressure(V_flow={0.0003,0.0006,0.0008},
    dp={45,35,15}*1000))
"Circulation pump";
```

This will model the following pump curve for the pump input signal  $y=1$ .



See [IBPSA.Fluid.Movers.Validation.PressureCurve](#) for a small example that validates the pressure curve specification.



# Best practice and modeling hints

# Building large system models

## 1. Understand the problem:

1. What question do you want to answer?
2. Know what you want to model.
  1. Draw system schematics.
  2. Identify control input.
  3. Draw the control loops.
  4. Determine the control sequences.

## 2. Compartmentalize:

Split the system into subcomponents that can be tested in isolation.

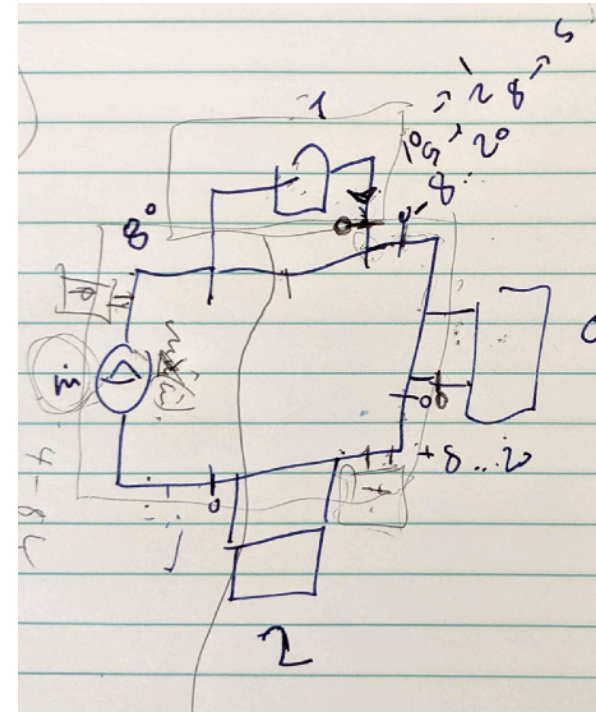
### 3. Implement:

Now, and only now, start implementing in software.

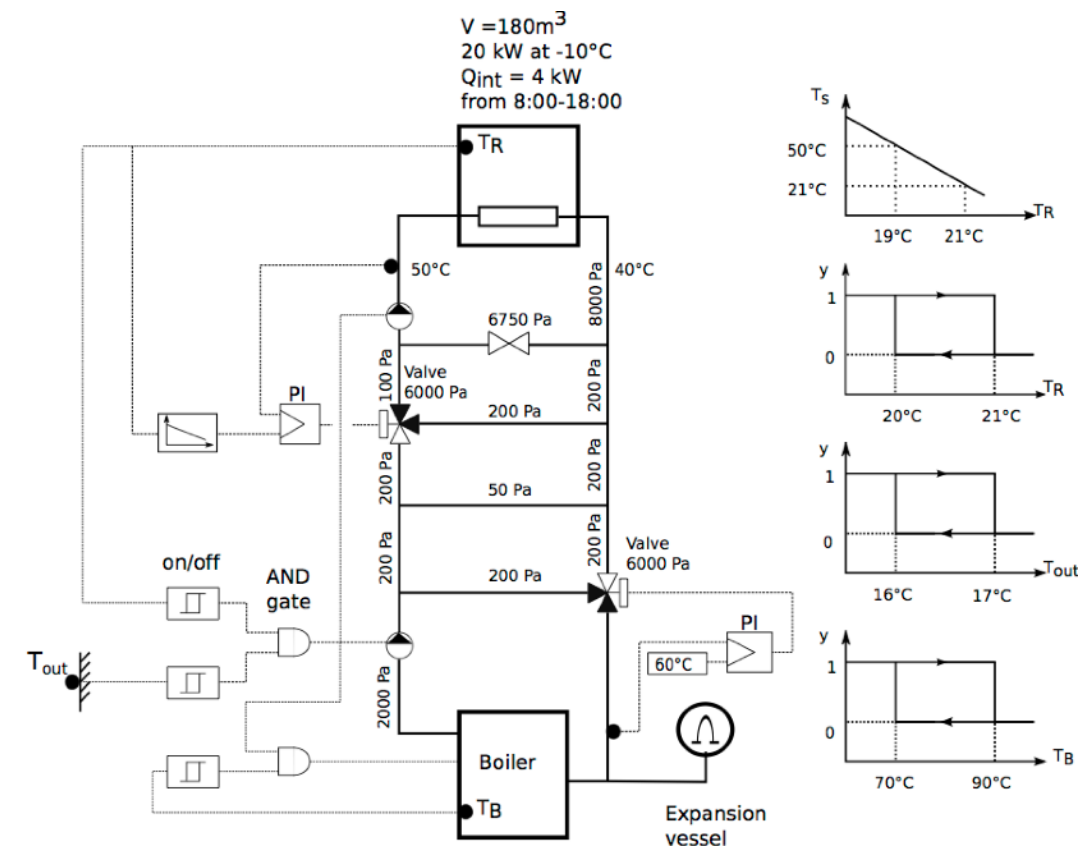
1. Document and build test cases as you go along.

Errors are easy to detect in small models, but hard in large models. If you add unit tests, you make sure what has been tested remains intact as the model evolves.

2. Assemble the subcomponents to build the full model.
3. Don't copy-paste models, you or your collaborator will regret it...  
Use version control, model instances, **extend**, **replaceable**, ...



*Iterate using hand-sketch of hydraulics and controls.*



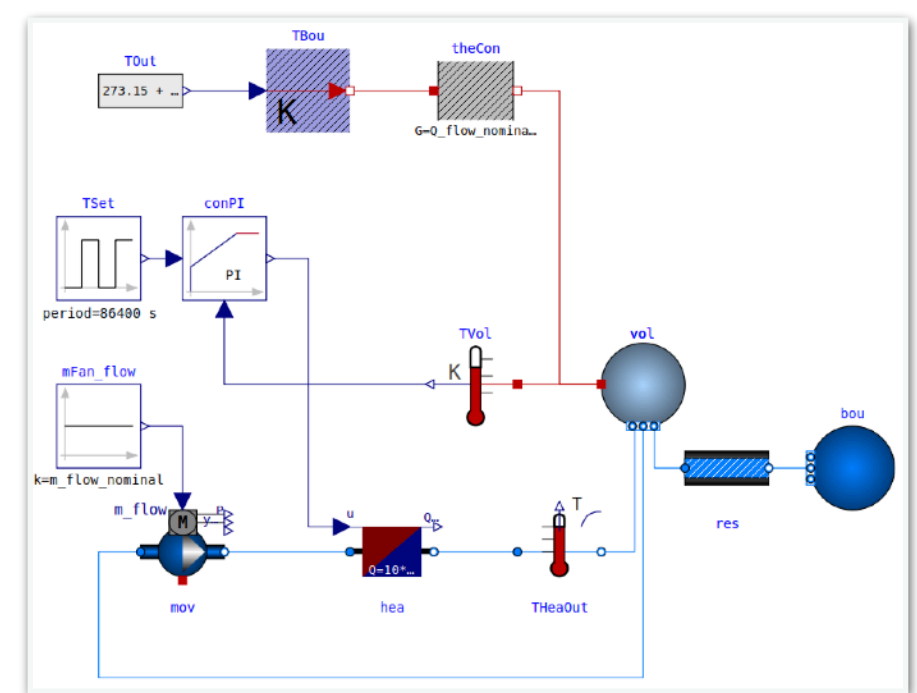
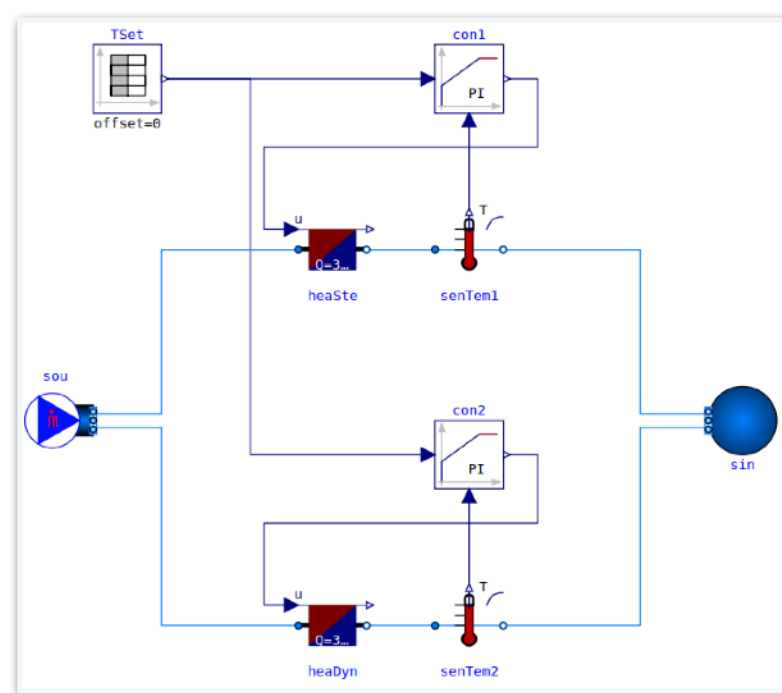
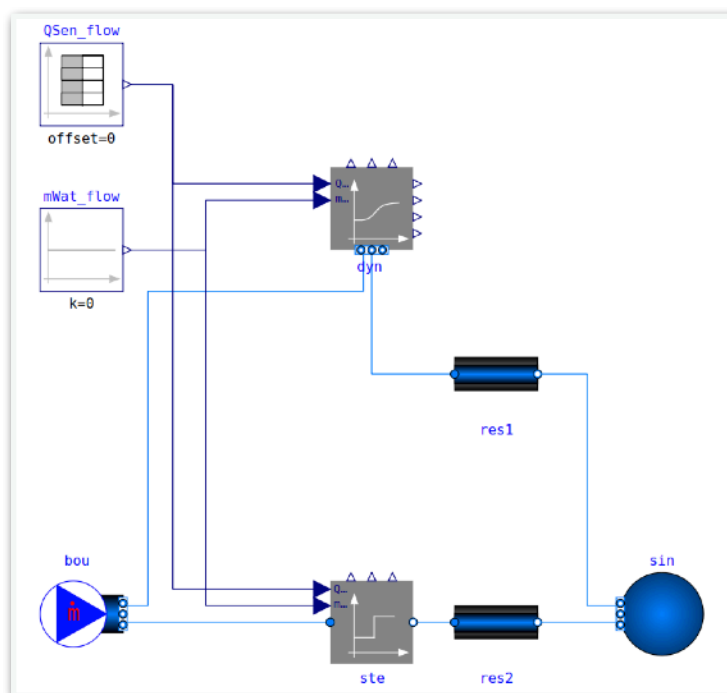
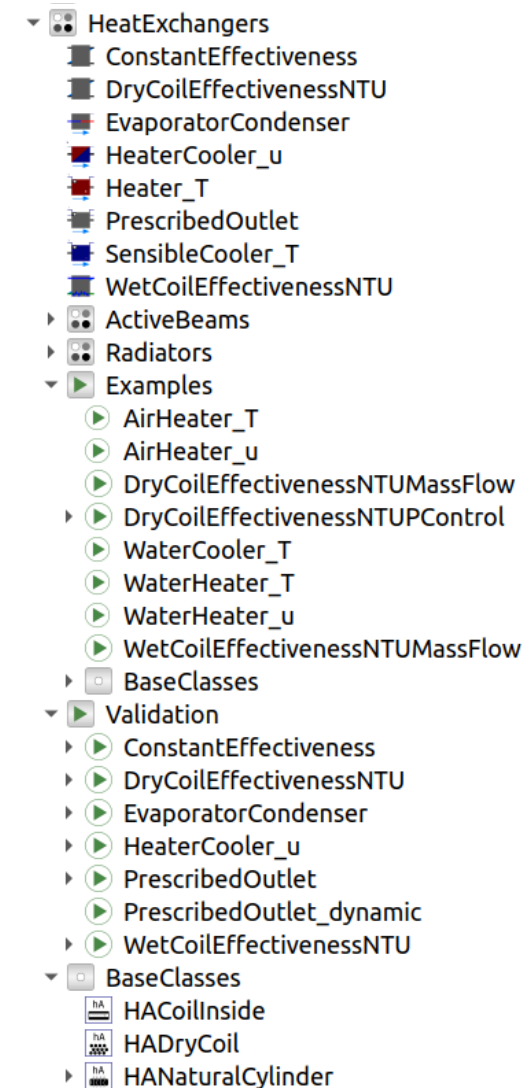
*Draw diagram, including control loops — even a hand-drawing saves time and increases quality.*

# Building large system models

How do you build and debug a large system model?

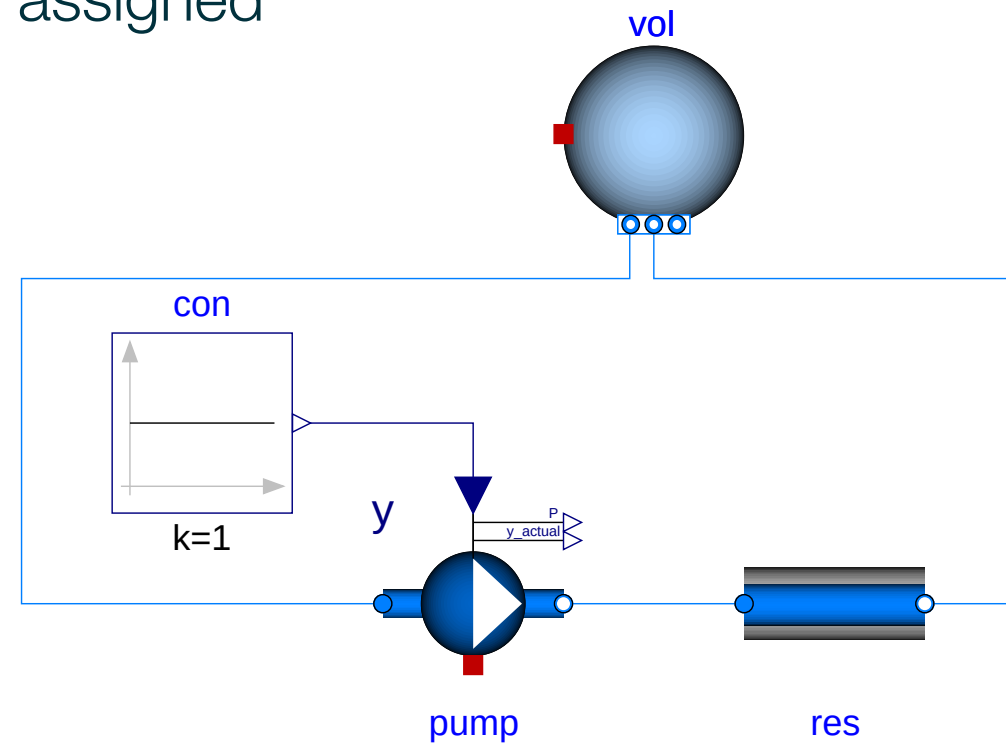
1. Split the model into small models — or better, architect the large model from the beginning to be based on smaller models
2. Test the smaller models for well known conditions.
3. Add smaller models to unit tests.

For example, see IBPSA.Fluid.HeatExchangers, in which each model contains a simple unit test, and components contain their own tests.

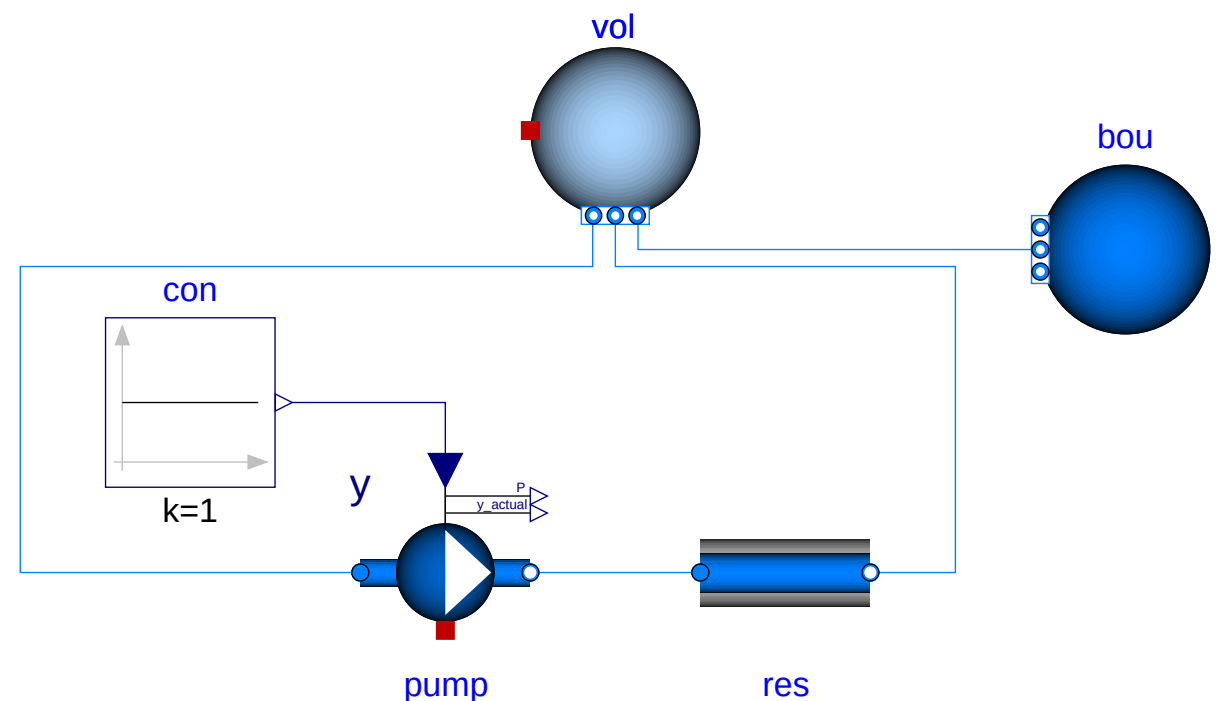


# All system models must have a reference pressure

Underdetermined model as no pressure state is assigned

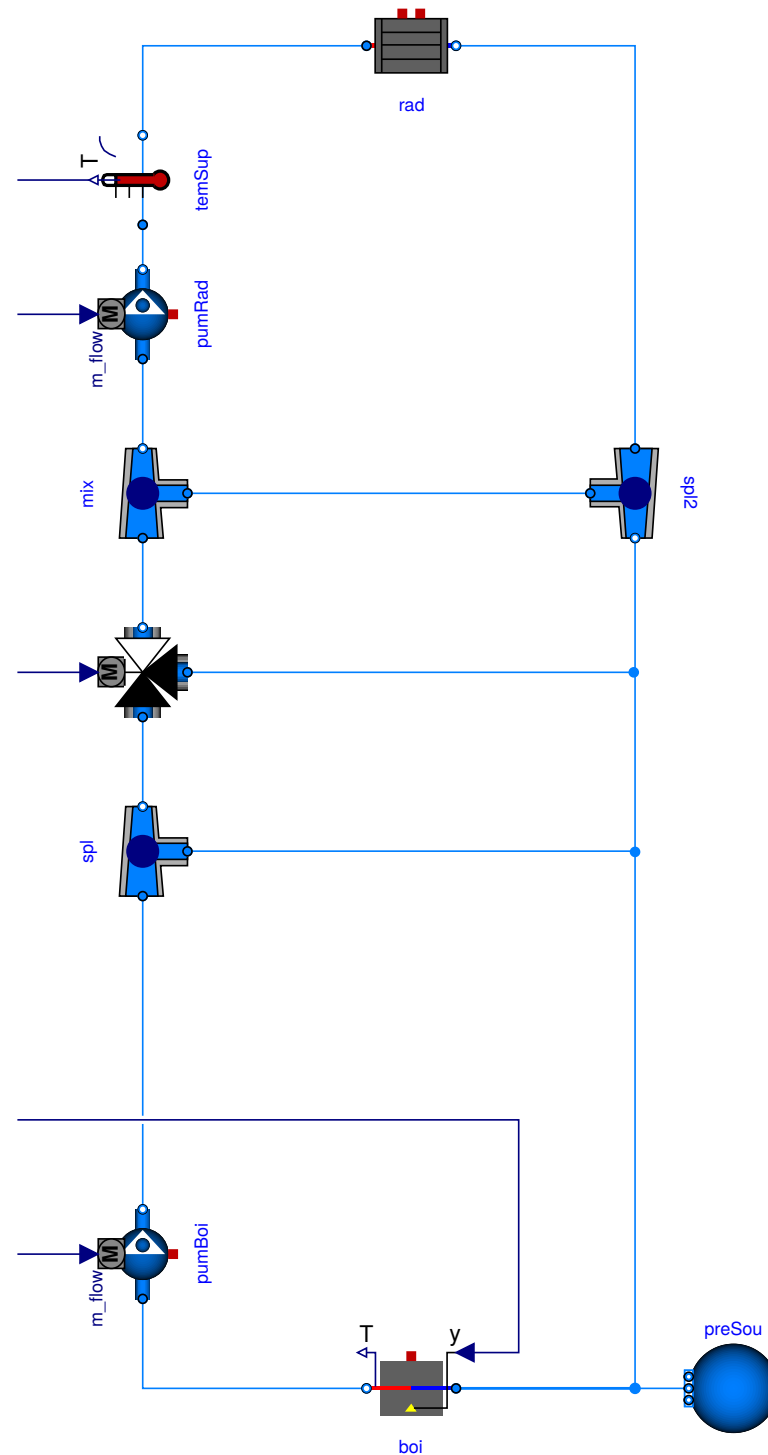


Model that provides a reference pressure through the instance **bou**.



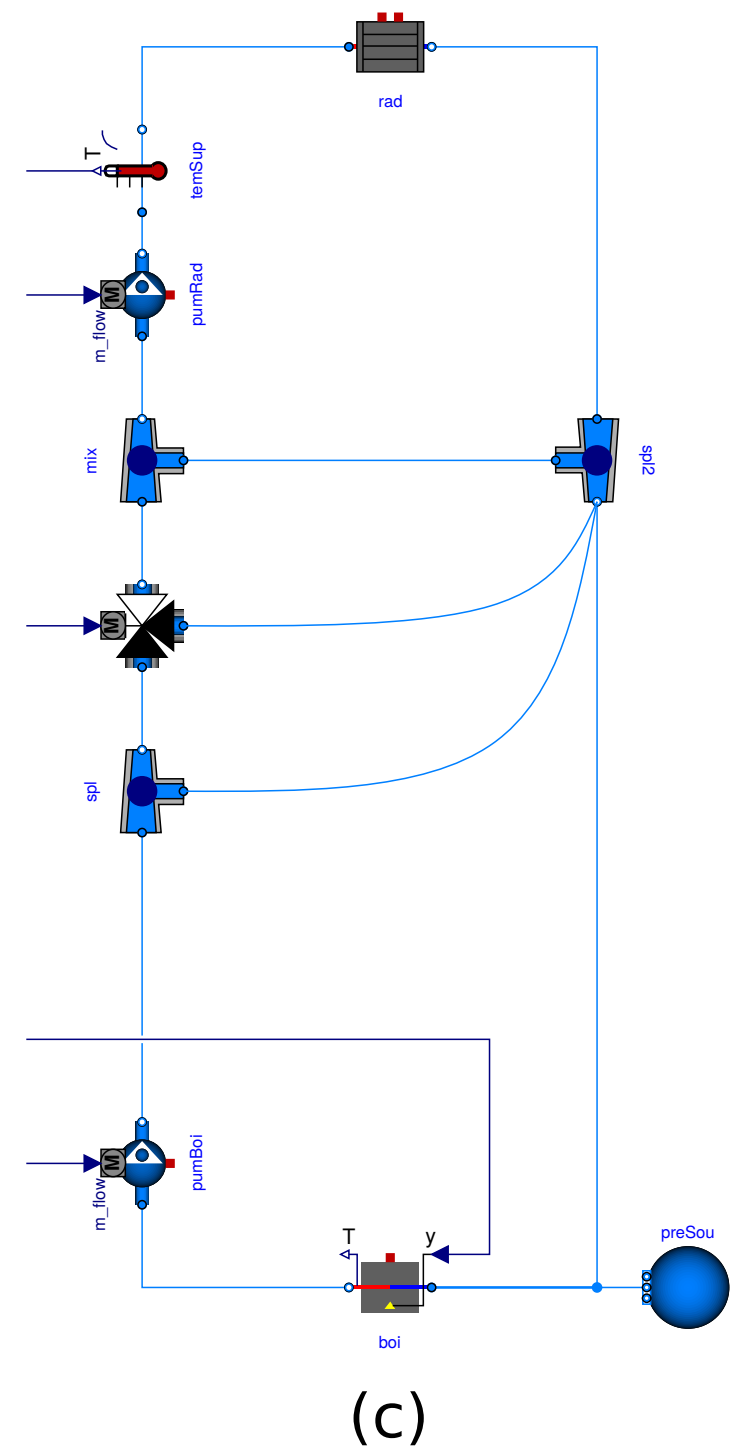
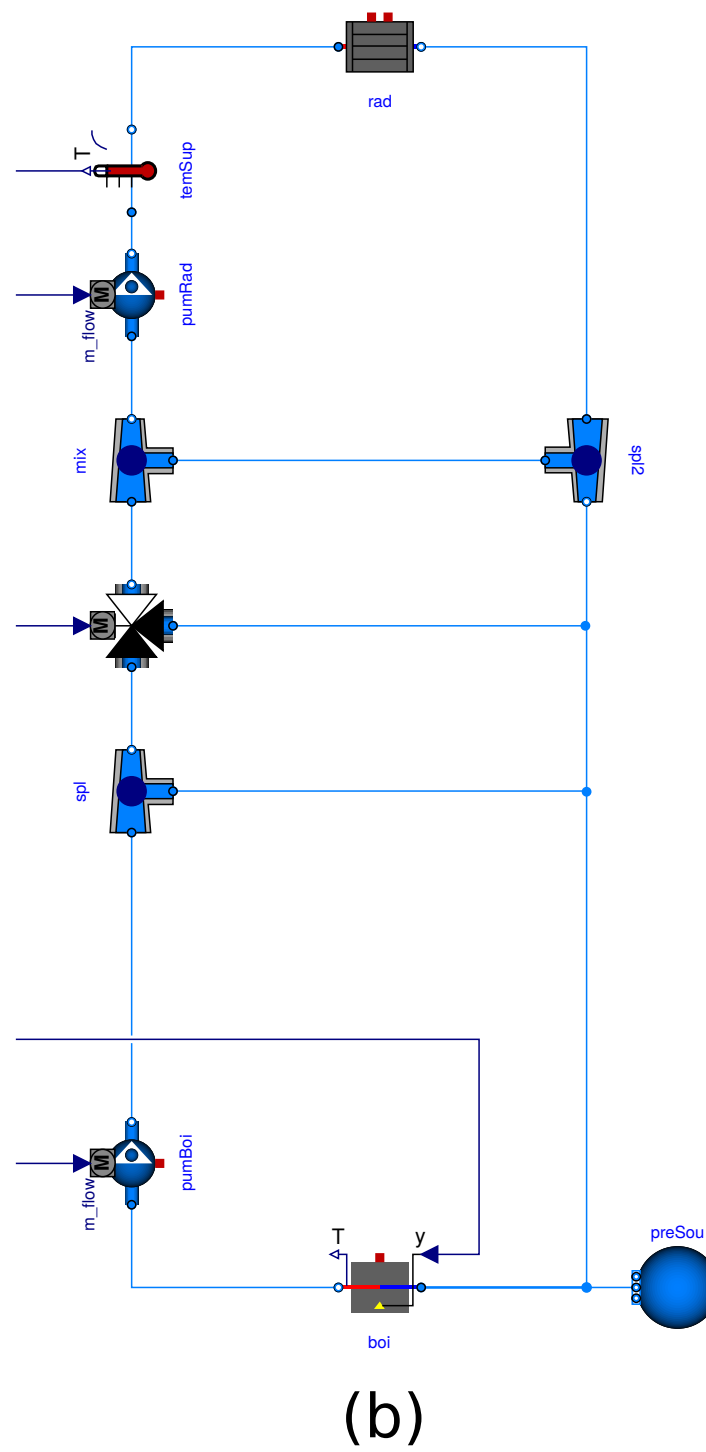
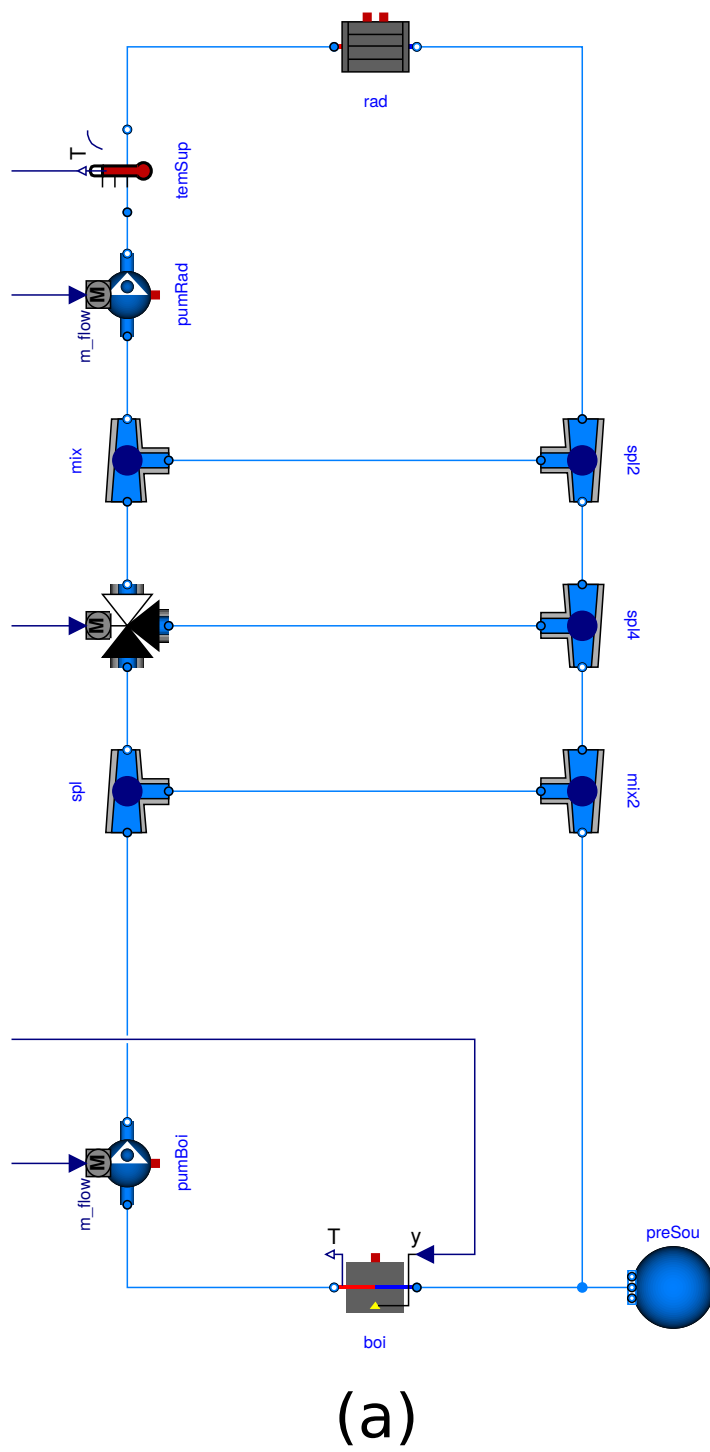
# Modeling of fluid junctions

What is wrong with this model?

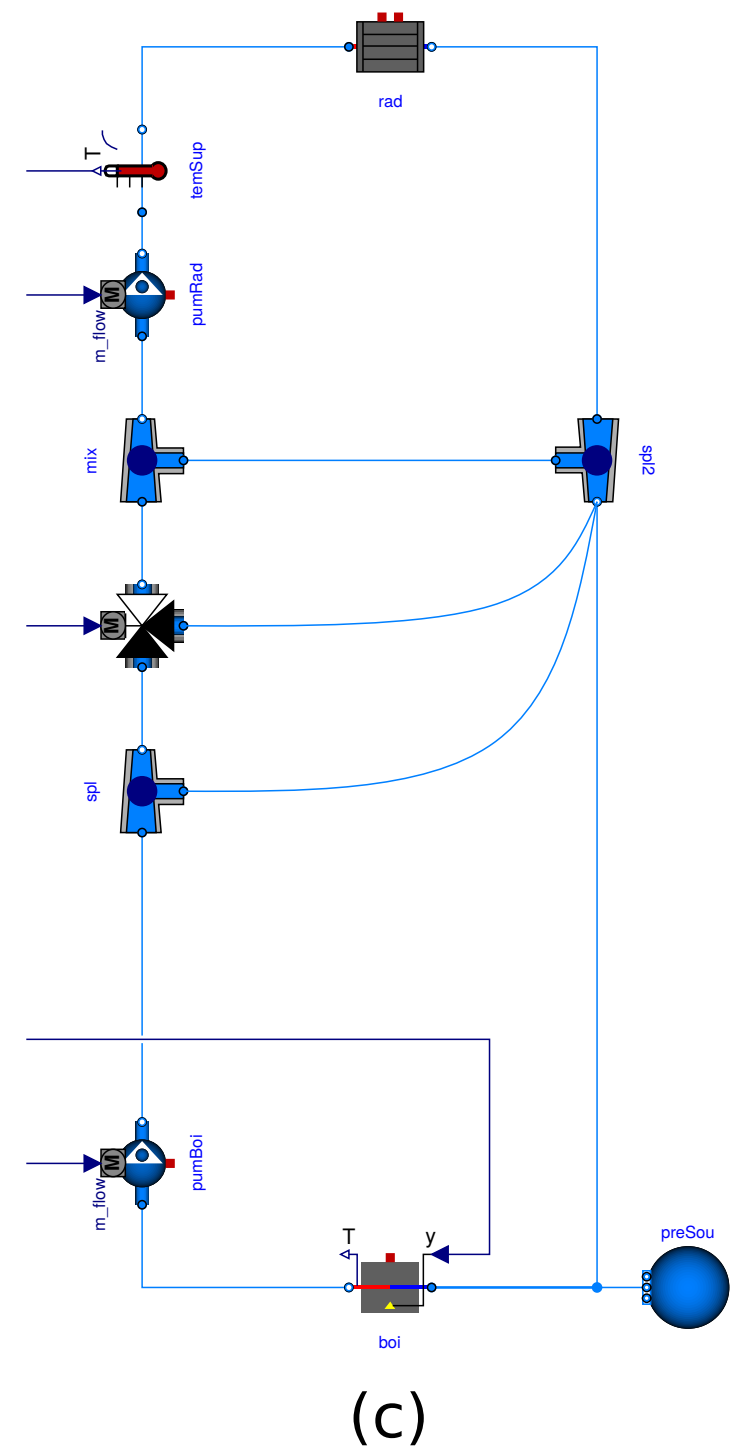
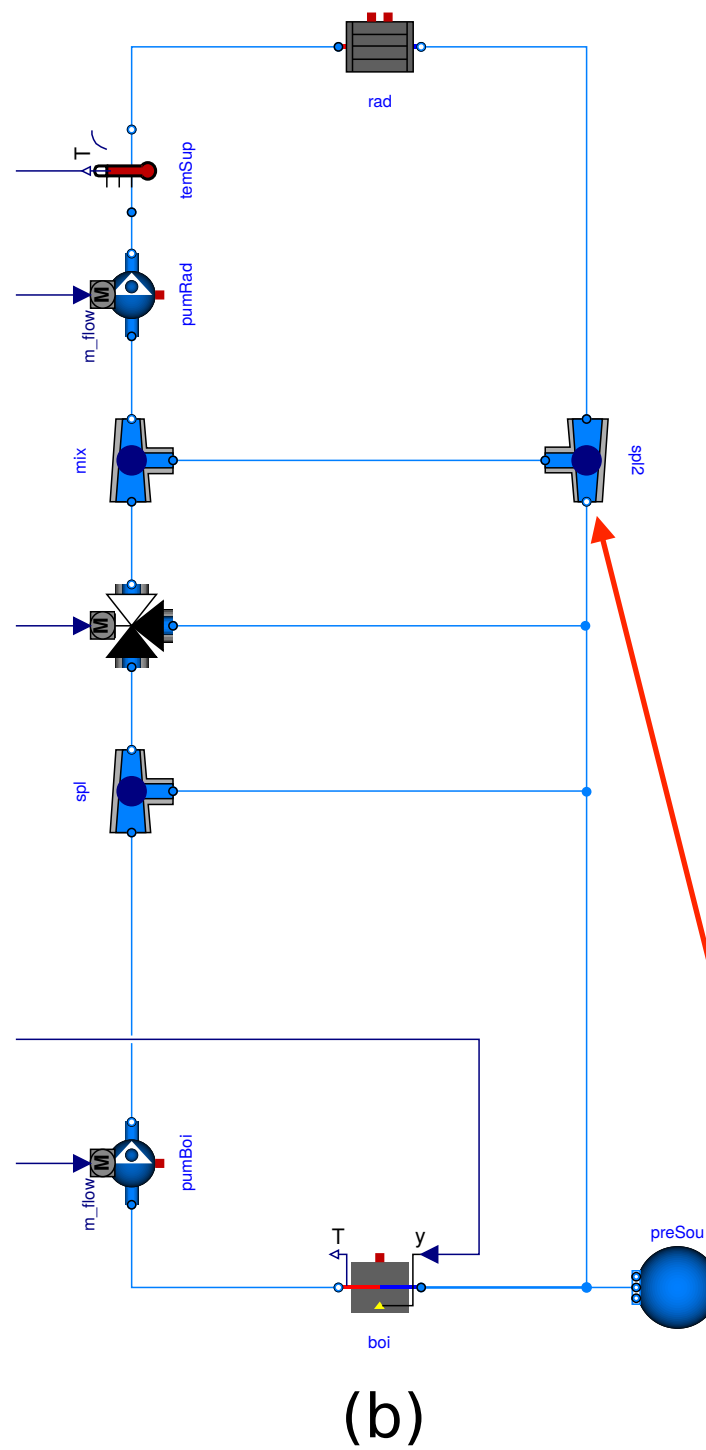
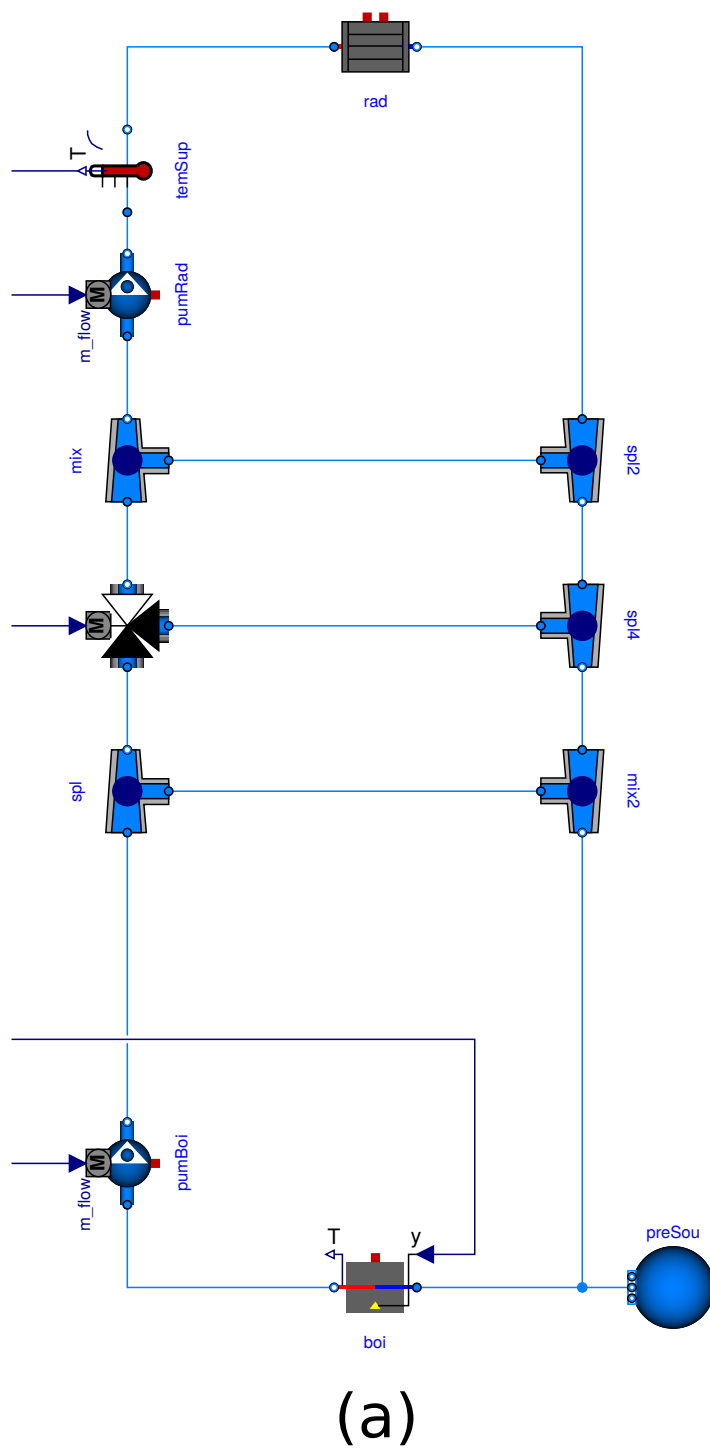




# Modeling of fluid junctions



# Modeling of fluid junctions

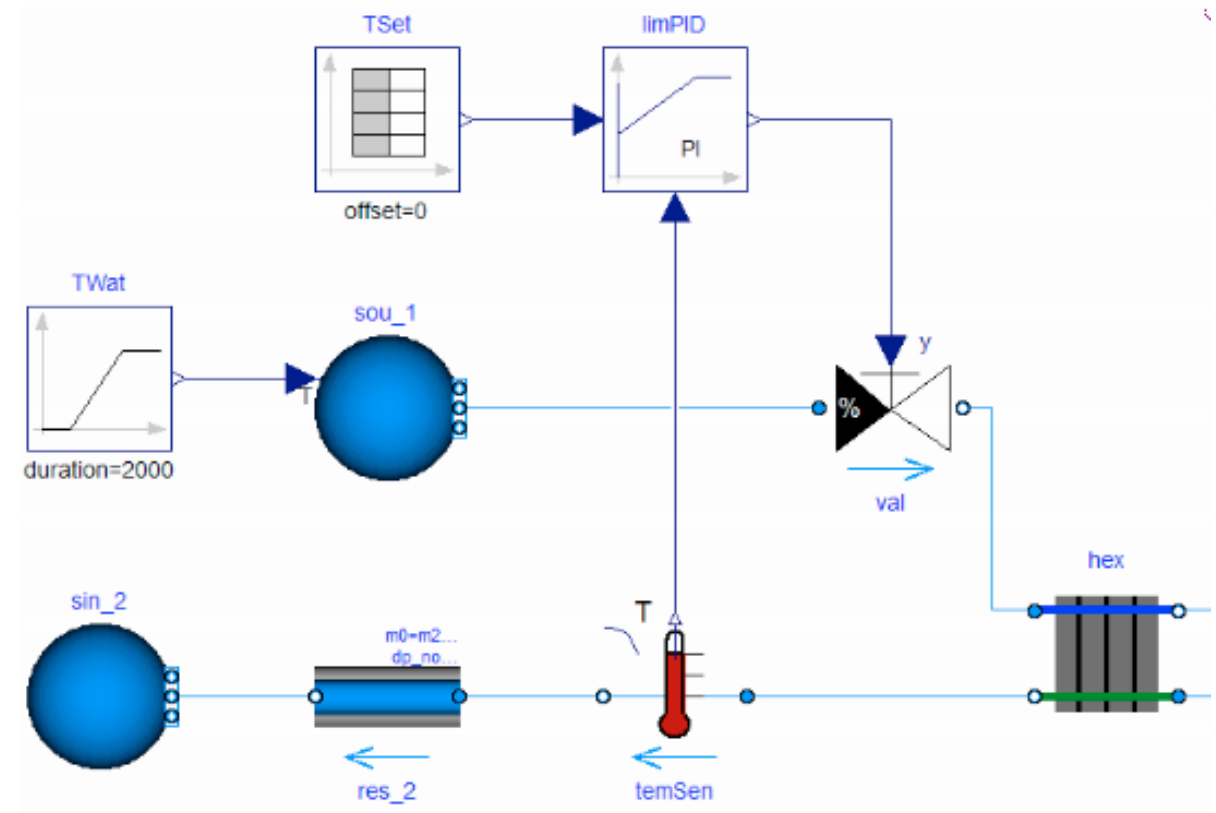


$$h = \frac{\sum_i \max(0, \dot{m}_i) h_i}{\sum_i \max(0, \dot{m}_i)}$$

# Avoid oscillations of sensor signal

Correct use because

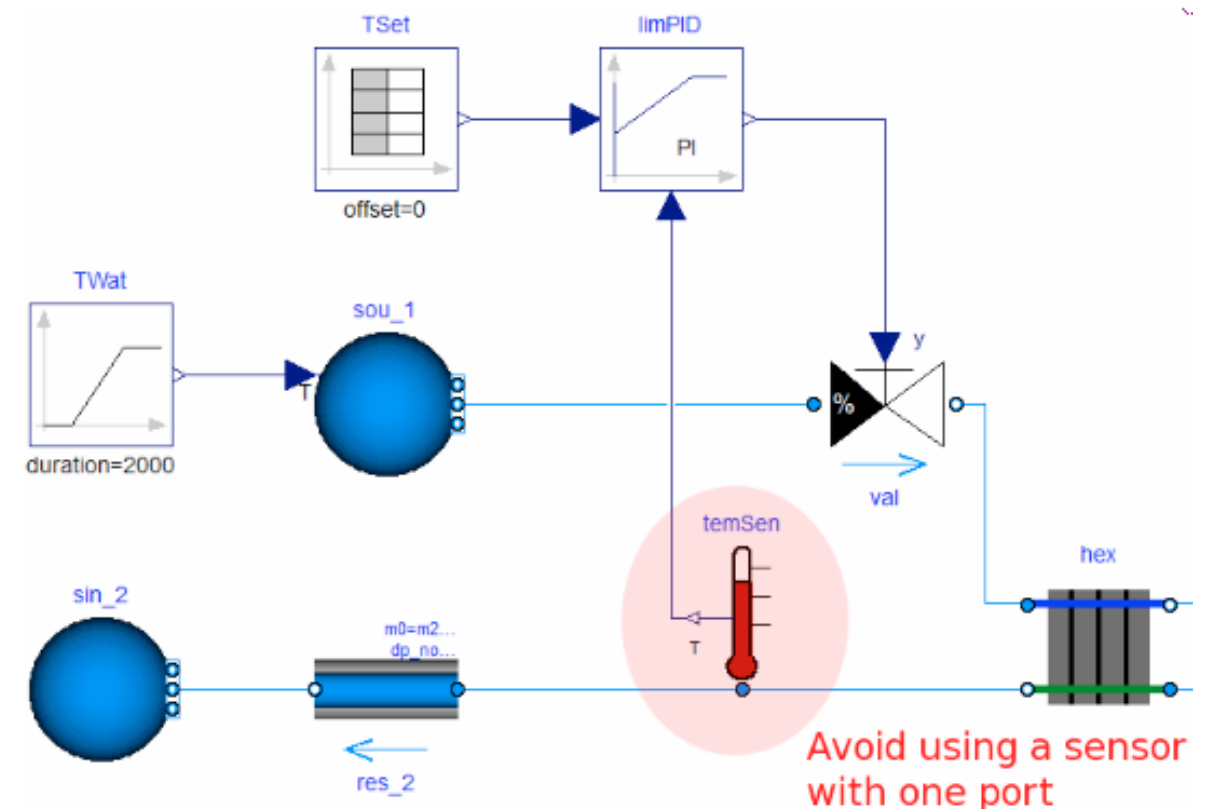
$$\tau \frac{dT}{dt} = \frac{|\dot{m}|}{\dot{m}_0} (\theta - T)$$



Incorrect, as sensor output oscillates if mass flow rate changes sign.

This happens for example if the mass flow rate is near zero and approximated by a solver.

See also Fluid.Sensors.UsersGuide

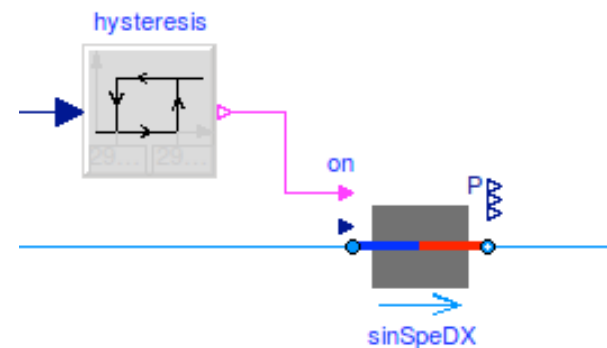


# Always guard against oscillations and noise (numerical noise or measurement noise)

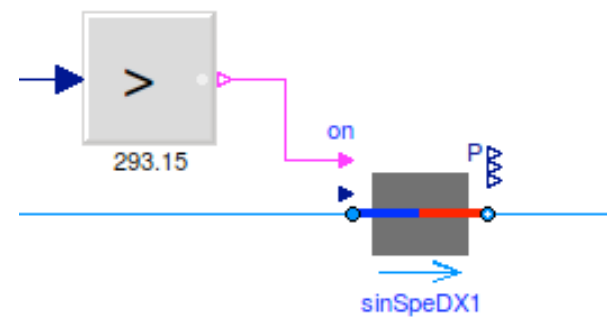
If the control input oscillates around zero, then this model can stall

What happens if this model is simulated with an adaptive time step?

Correct configuration



Avoid this configuration



```
model Test
  Real x(start=0.1);
equation
  der(x) = if x > 0 then -1 else 1;
end Test;
```

# Don't Repeat Yourself: Propagate common parameters

Don't assign the same values to multiple parameters:

```
Pump pum(m_flow_nominal=0.1) "Pump";  
TemperatureSensor sen(m_flow_nominal=0.1) "Sensor";
```

Instead, propagate parameters and assign the value once:

```
Modelica.SIunits.MassFlowRate m_flow_nominal = 0.1  
  "Nominal mass flow rate";  
Pump pum(final m_flow_nominal=m_flow_nominal) "Pump";  
TemperatureSensor sen(final m_flow_nominal=m_flow_nominal) "Sensor";
```

Assignments can include computations, such as

```
Modelica.SIunits.HeatFlowRate QHea_nominal = 3000  
  "Nominal heating power";  
Modelica.SIunits.TemperatureDifference dT = 10  
  "Nominal temperature difference";  
Modelica.SIunits.MassFlowRate m_flow_nominal = QHea_nominal/dT/4200  
  "Nominal mass flow rate";
```

...



# Don't Repeat Yourself:

Always define the media at the top-level

Top-level system-model

```
replaceable package Medium = IBPSA.Media.Air  
  "Medium model";
```

Propagate medium to instance of model

```
TemperatureSensor sen(  
  redeclare final package Medium = Medium,  
  final m_flow_nominal=m_flow_nominal) "Sensor";
```

Note: For arrays of parameters, use the **each** keyword, as in

```
TemperatureSensor sen[2](  
  each final m_flow_nominal=m_flow_nominal)  
  "Sensor";
```

Questions?