

# Appendix A

## Calculation of the Number of Degrees of Freedom of Robots with Closed Chains

### A.1 Introduction

Let us recall first that the mobility, or number of *DOF*, of a robot is defined as the number of independent joint variables required to specify the location of all the links of the robot in space. It is equal to the minimal number of actuated joints to control the system.

The number of degrees of freedom  $N_{dof}$  of a robot is equal to the number of joints in the case of tree structure system  $L$ . In the case of a closed-loop mechanism, the calculation of the mobility  $N_{dof}$  can be expressed by the following relation:

$$N_{dof} = L - c \quad (\text{A.1})$$

where  $L$  is the number of joints of the structure and  $c$  is the number of independent relationships (constraints) between the joint variables, i.e. the number of dependent joints.

Since 1854 with the work of P.A. Chebychev, several researchers have proposed different formulas that can be used to find the mobility of complex systems. Recently, Gogu (2008) has evaluated 35 methods that have been proposed to calculate the mobility of complex systems. He concluded that the majority of methods cannot properly calculate mobility for all mechanisms, and only those that require the construction of the kinematic constraint equations can give a good result.

In case of a single loop,  $c$  represents the number of independent kinematic constraint equations of the loop. Consequently  $c \leq 6$  for a spatial loop and  $c \leq 3$  for a planar loop. Consequently,  $N_{dof}$  gives the dimension of the space in which the situation of all the links belong. It is possible to calculate it by calculating the maximum rank of the Jacobian matrix of the serial structure constructed by cutting one link in the loop. This result can be interpreted by the fact that the open structure has  $L$  degrees of freedom, and since the  $c$  degrees of freedom of the terminal link will be lost when closing the loop, thus the number of remaining degrees of freedom is equal to  $L - c$ .

In case of a system composed of  $B$  independent closed loops, the mobility of the system may be calculated by:

$$N_{dof} = L - \sum_{j=1}^B c_j. \quad (\text{A.2})$$

This simple formula gives good results for most robot structures but it can yield bad results for certain complex systems and does not give information about the type of motion of the system. However, for some robots, the exact solution is obtained by analyzing the kinematic constraints and taking into account the coupling between the loops (Hervé 1978; Le Borzec and Lotterie 1975). In the following, we present two methods: the Morokine's method and the Gogu's method.

## A.2 Moroskine's Method

The mobility can be calculated correctly using the rank of the matrix  $\mathbf{J}_c$  defined in the following equation:

$$c = \max_{\mathbf{q}}(\text{rank}(\mathbf{J}_c(\mathbf{q}))) \quad (\text{A.3})$$

where  $\mathbf{J}_c$  is the Jacobian of the constraint equations between the joint variables such that:

$$\mathbf{J}_c(\mathbf{q})\dot{\mathbf{q}} = \mathbf{0}. \quad (\text{A.4})$$

$\mathbf{J}_c(\mathbf{q})$  can be calculated by derivation of the geometric constraint equations of the loops or by constructing the constraint equations of the velocities through the loop. In fact, from this equation, we deduce that  $\dot{\mathbf{q}}$  belongs to the null space of  $\mathbf{J}_c$ . Therefore, at a given configuration, the number of degrees of freedom is equal to the dimension of the null space of  $\mathbf{J}_c$ . Consequently:

$$N_{dof} = \min_{\mathbf{q}}(\dim(\mathcal{N}(\mathbf{J}_c(\mathbf{q})))) \quad (\text{A.5})$$

where  $\mathcal{N}(\mathbf{J}_c(\mathbf{q}))$  is the null space of the matrix  $\mathbf{J}_c$ .

In general, the rank of  $\mathbf{J}_c(\mathbf{q})$  must be calculated for  $\mathbf{q}$  satisfying the closure constraints of the loops. For a single closed loop, the rank can be calculated using random values of its joints.

This method yields the correct result, but it is significantly more difficult to execute. In order to find the rank of the Jacobian matrix defining a mechanism with closed loops, the kinematic constraint equations must be solved. The kinematic constraint equations display the relationship between the joint variables in the mechanism in

order to ensure loop closure. In general it is difficult to solve these equations symbolically though it is possible to solve them numerically in order to obtain some random configurations satisfying the closure conditions.

### A.3 Gogu's Method

In order to overcome the drawbacks of the previous method, Gogu proposed a method that does not require the construction of the kinematic constraint equations, but will yield the correct mobility for all mechanisms including complex parallel mechanisms. We present here how to use Gogu's method to calculate the mobility for:

- single loop kinematic chains,
- parallel mechanisms with simple legs, and
- parallel mechanisms with complex legs.

The proposed solution makes use of the mobility of the terminal link of simple open loop which is equal to the rank of the Jacobian matrix between the terminal link or the mobile platform in the case of a *PKM* and the base. In fact the mobility  $M_{n/0}$  of the terminal link with respect to the base of an open loop chain is equal to the dimension of the task (Cartesian) space  $\dim(E(\mathbf{x}))$ .

$$M_{n/0} = M = \text{rank}(\mathbf{J}_n) = \dim(E(\mathbf{x})). \quad (\text{A.6})$$

#### A.3.1 Mobility of Single Loop Kinematic Chains

A link must be opened to obtain an equivalent simple open loop. If the number of joints is equal to  $L$ , and the chain is opened around joint  $L$  such that the links  $1, \dots, L$  constitute a serial structure, then the rank of the Jacobian matrix  $\mathbf{J}_L$  gives the number of joint variables that lose their independence after loop closure. Thus the mobility of the closed loop is given by:

$$N_{dof} = L - \text{rank}(\mathbf{J}_L) \quad (\text{A.7})$$

where  $\text{rank}(\mathbf{J}_L) = M_{L/0}$  is the mobility of link  $L$  w.r.t. link 0 in the open chain.

A second method to calculate the mobility is to open the structure around a joint  $k < L$ , in order to obtain two serial branches with  $n_1$  and  $n_2$  joints respectively.

Supposing  $\text{rank}(\mathbf{J}_{n_i}) = \dim(E(\mathbf{x}_i))$  is the dimension of the task (Cartesian) space of branch  $i$ , with  $i = 1, 2$ , and  $\dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2))$  gives the dimension of the common task space that the two branches share. Thus, the number  $c$  of joints losing their independence will be obtained as follows:

$$\begin{aligned} c &= \text{rank}(\mathbf{J}_L) = \dim(E(\mathbf{x}_1)) + \dim(E(\mathbf{x}_2)) - \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2)) \\ &= \text{rank}(\mathbf{J}_{n_1}) + \text{rank}(\mathbf{J}_{n_2}) - \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2)). \end{aligned} \quad (\text{A.8})$$

The mobility can be calculated using  $N_{dof} = L - c$ .

If there is no kinematic redundancies such that  $\text{rank}(\mathbf{J}_{n_1}) = N_1$  and  $\text{rank}(\mathbf{J}_{n_2}) = N_2$ , the mobility of the mechanism is given by:

$$N_{dof} = L - \text{rank}(\mathbf{J}_L) = \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2)). \quad (\text{A.9})$$

In case of redundancy, the mobility will be:

$$N_{dof} = \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2)) + \text{number of redundant joints in both branches}. \quad (\text{A.10})$$

### A.3.2 Mobility of Parallel Mechanisms with Serial Legs

Let us consider a parallel mechanism with a base platform and a mobile platform that are connected together with  $m$  simple open kinematic chains. The number of joints of each chain is denoted by  $n_i$  for  $i = 1, \dots, m$ . The mobility of the platform  $M$  with respect to the base is given by the dimension of the common task spaces of the simple legs associated with the parallel mechanism, as seen in the following equations as long as there are no redundancy:

$$N_{P/0} = \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2) \cdots \cap E(\mathbf{x}_m)). \quad (\text{A.11})$$

The number of joints that lose their independence after loop closure is equal to the difference between the sum of mobilities of the terminal links of simple chains and the mobility of the platform:

$$c = \sum_{j=1}^m \dim(E(\mathbf{x}_j)) - \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2) \cdots \cap E(\mathbf{x}_m)). \quad (\text{A.12})$$

Thus the mobility of the structure is given by

$$N_{dof} = \sum_{j=1}^m n_j - \sum_{j=1}^m \dim(E(\mathbf{x}_j)) + \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2) \cdots \cap E(\mathbf{x}_m)) \quad (\text{A.13})$$

where  $\sum_{j=1}^m n_j = L$ , the number of total joints.

Note that in case of non-redundant legs,  $n_j = \dim(E(\mathbf{x}_j))$ , thus leading to mobility as the dimension of the common space between all the legs.

### A.3.3 Mobility of Parallel Mechanisms with Complex Legs

A parallel mechanism with complex legs is a complex mechanism with  $L$  joints in which the mobile platform is connected to the base by  $m \geq 2$  legs, of which at least one leg contains at least one closed loop. Theoretically in this case the platform is not uniquely defined, but in practice it is easy to select an appropriate one. The mobility of the mechanism in this case is calculated by:

$$N_{dof} = L - \sum_{j=1}^m \dim(E(\mathbf{x}_j)) + \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2) \dots \cap E(\mathbf{x}_m)) - C_{cl} \quad (\text{A.14})$$

where  $C_{cl} = \sum_{k=1}^{n_c} c_k$  is sum of the additional joint variables that lose their independence in the closed loops belonging to the complex legs.  $C_{cl}$  can be calculated using the previous cases in Sects. A.3.1 and A.3.2 depending on whether the leg contains loops connected serially or in parallel respectively.

## A.4 Examples

In this section we calculate the mobility of some robot architectures treated in the different chapters of this book.

### A.4.1 The Planar Five-Bar Mechanism

This structure is shown in Fig. 7.3. It consists of five revolute joints with parallel axes. The system ensures motion in the plane perpendicular to the joint axes. The mobility can be calculated using different interpretation:

1. The number of independent constraint equations around the loop is equal to 3, specifying the equality of  $x$ ,  $y$  and the orientation  $\phi$ . Thus, the mobility is equal to 2.
2. We open the loop at the terminal joint  $L$ , to obtain a serial architecture with five revolute joints. It is intuitive to deduce that the rank of this serial system is equal to the mobility of its terminal link, i.e. 3. Thus, connecting the terminal link with the base leads to a loss of these three degrees of freedom. Consequently, the mobility of the system is equal to 2.
3. We open the structure to obtain two serial chains connected with the base, such that one chain contains 3 joints and the other contains 2 joints. The dimension of the terminal link spaces of these chains are respectively 3 and 2. The motion of the first chain can be classified as: Tx (translation along  $\mathbf{x}_0$ ), Ty (translation along  $\mathbf{y}_0$ ) and Rz (rotation about  $\mathbf{z}_0$ ), whereas the motion of the terminal link of

the second chain, whose mobility is equal to 2, can be any two degrees of freedom among: Tx, Ty and Rz.

Consequently the common motion when connecting the two terminal links together will be of dimension 2, and can be represented by any two variables among (Tx, Ty, Rz). Thus the number of degrees of freedom of the terminal link (point) and the number of degrees of freedom of the system are respectively:

$$n_{dof} = 2$$

$$N_{dof} = L - \sum_{j=1}^2 \dim(E(\mathbf{x}_j)) + \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2)) = 2.$$

Consequently, two axes should be actuated. We selected the actuators near the base. In case of actuating more than two joints, the system will be redundantly actuated.

#### ***A.4.2 The Planar 3-RPR Parallel Robot***

This *PKM* is shown in Fig. 7.5. It has three legs, each being composed of three joints (*R*, *P*, and *R* joints) ensuring planar motions.

The mobility of each leg is composed of Tx, Ty and Rz.

Thus

$$E(\mathbf{x}_i) = \text{Tx, Ty, Rz for } i = 1, 2, 3$$

and thus the common space consists of: Tx, Ty and Rz.

Consequently, the mobility is equal to 3. The three prismatic joints are selected to be the actuated joints.

#### ***A.4.3 The Orthoglide***

The Orthoglide (Fig. 7.7) is composed of three legs, each of them having a prismatic joint and a spatial parallelogram. The prismatic joints are perpendicular. The mobility of the Orthoglide will be analyzed using the equivalent kinematic chain of each leg, which is considered to be composed of *PUU* architecture, with *U* a universal joint represented by two intersecting revolute axes. Thus the motion of terminal link of each leg is composed of a prismatic joint, then two rotational and two translation

degrees of freedom perpendicular on the prismatic axis of the first joint. The mobility of the three legs can be written respectively as:

$$E(\mathbf{x}_1) = T_z, R_x, R_y, T_x, T_y$$

$$E(\mathbf{x}_2) = T_x, R_y, R_z, T_y, T_z$$

$$E(\mathbf{x}_3) = T_y, R_x, R_y, T_x, T_z.$$

Thus

$$n_{dof} = \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2) \cap E(\mathbf{x}_3)) = 3$$

$$E(\mathbf{x}_p) = T_x, T_y, T_z$$

$$N_{dof} = L - \sum_{j=1}^3 \dim(E(\mathbf{x}_j)) + \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2) \cap E(\mathbf{x}_3)) = 3.$$

We selected the three prismatic joints as actuators.

#### A.4.4 The Tripteron

This *PKM* is shown in Fig. 9.3. It is composed of three legs, each of them having a serial architecture with 4 joints. The first joint is prismatic and the other joints are revolute. All the *P* and *R* joints of the same leg have parallel axes.

We can easily deduce that:

$$E(\mathbf{x}_1) = T_x, T_y, T_z, R_x$$

$$E(\mathbf{x}_2) = T_x, T_y, T_z, R_y$$

$$E(\mathbf{x}_3) = T_x, T_y, T_z, R_z.$$

Thus

$$n_{dof} = \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2) \cap E(\mathbf{x}_3)) = 3$$

$$E(\mathbf{x}_p) = T_x, T_y, T_z$$

$$N_{dof} = L - \sum_{j=1}^3 \dim(E(\mathbf{x}_j)) + \dim(E(\mathbf{x}_1) \cap E(\mathbf{x}_2) \cap E(\mathbf{x}_3)) = 3.$$

We selected the three prismatic joints as actuators.

**Remarks**

Note that the simple relation of (A.2) can give the correct result for the first two examples (planar five-bar mechanism, and 3- $R\underline{P}R$ ), but it does not give the correct result for the Orthoglide nor for the Tripteron.



## Appendix B

### Lagrange Equations with Multipliers

Let us consider a mechanical system whose Lagrangian  $L$  can be computed by the knowledge of the generalized coordinates  $\mathbf{q}$  (and  $\dot{\mathbf{q}}$ ). Let us assume that  $\mathbf{q}$  groups two sets of variables  $\mathbf{q}_a$  and  $\mathbf{q}_d$  ( $\mathbf{q}^T = [\mathbf{q}_a^T \ \mathbf{q}_d^T]$ ) and  $\dot{\mathbf{q}}^T = [\dot{\mathbf{q}}_a^T \ \dot{\mathbf{q}}_d^T]$ ) which are not independent and are related through the expressions:

$$\mathbf{h}(\mathbf{q}_a, \mathbf{q}_d) = \mathbf{0} \quad (\text{B.1})$$

and:

$$\mathbf{A}(\mathbf{q}_a, \mathbf{q}_d)\dot{\mathbf{q}}_d + \mathbf{B}(\mathbf{q}_a, \mathbf{q}_d)\dot{\mathbf{q}}_a = \mathbf{0} \quad (\text{B.2})$$

where  $\mathbf{A}$  and  $\mathbf{B}$  are two matrices depending on  $\mathbf{q}_a$  and  $\mathbf{q}_d$ :

$$\mathbf{A}(\mathbf{q}_a, \mathbf{q}_d) = \left[ \frac{\partial \mathbf{h}(\mathbf{q}_a, \mathbf{q}_d)}{\partial \mathbf{q}_d} \right] \quad (\text{B.3})$$

and

$$\mathbf{B}(\mathbf{q}_a, \mathbf{q}_d) = \left[ \frac{\partial \mathbf{h}(\mathbf{q}_a, \mathbf{q}_d)}{\partial \mathbf{q}_a} \right]. \quad (\text{B.4})$$

Moreover, we consider that the variables  $\mathbf{q}_a$  are some variables corresponding (in the frame of this book) to motor coordinates, motors which are exerting some input efforts  $\boldsymbol{\tau}$  on the system. This is not the case for the coordinates  $\mathbf{q}_d$ .

The usual Lagrange equations (6.1) cannot be derived because all coordinates in  $\mathbf{q}$  are not independent. In order to modify the Lagrange equations (6.1) to take into account the constraints (B.1) and (B.2), we must include some generalized constraint forces  $\boldsymbol{\gamma}_a$  and  $\boldsymbol{\gamma}_d$  such that:

$$\begin{aligned}\boldsymbol{\tau} + \boldsymbol{\gamma}_a &= \boldsymbol{\tau}_a, \text{ where } \boldsymbol{\tau}_a = \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\mathbf{q}}_a} \right)^T - \left( \frac{\partial L}{\partial \mathbf{q}_a} \right)^T \\ \boldsymbol{\gamma}_d &= \boldsymbol{\tau}_d, \text{ where } \boldsymbol{\tau}_d = \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\mathbf{q}}_d} \right)^T - \left( \frac{\partial L}{\partial \mathbf{q}_d} \right)^T.\end{aligned}\tag{B.5}$$

Of course, these generalized constraint forces  $\boldsymbol{\gamma}_a$  and  $\boldsymbol{\gamma}_d$  are internal to the system and produce no work, i.e. the *PVP* states that we have, for any arbitrary velocities  $\dot{\mathbf{q}}_a^*$  and  $\dot{\mathbf{q}}_d^*$ ,

$$\dot{\mathbf{q}}_d^{*T} \boldsymbol{\gamma}_d + \dot{\mathbf{q}}_a^{*T} \boldsymbol{\gamma}_a = 0.\tag{B.6}$$

Now, taking the transposed expression of (B.2), we obtain

$$\dot{\mathbf{q}}_d^T \mathbf{A}^T + \dot{\mathbf{q}}_a^T \mathbf{B}^T = \mathbf{0}.\tag{B.7}$$

This expression is also true if we right-multiply it by any arbitrary vector  $\boldsymbol{\lambda}$ :

$$\dot{\mathbf{q}}_d^T \mathbf{A}^T \boldsymbol{\lambda} + \dot{\mathbf{q}}_a^T \mathbf{B}^T \boldsymbol{\lambda} = 0.\tag{B.8}$$

By identification between (B.6) and (B.8), we can see that we have:

$$\boldsymbol{\gamma}_a = \mathbf{B}^T \boldsymbol{\lambda}\tag{B.9}$$

and

$$\boldsymbol{\gamma}_d = \mathbf{A}^T \boldsymbol{\lambda}\tag{B.10}$$

from which we obtain the new set of Lagrange equations, in which  $\boldsymbol{\lambda}$  is called the vector of Lagrange multipliers:

$$\begin{aligned}\boldsymbol{\tau} + \mathbf{B}^T \boldsymbol{\lambda} &= \boldsymbol{\tau}_a, \text{ where } \boldsymbol{\tau}_a = \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\mathbf{q}}_a} \right)^T - \left( \frac{\partial L}{\partial \mathbf{q}_a} \right)^T \\ \mathbf{A}^T \boldsymbol{\lambda} &= \boldsymbol{\tau}_d, \text{ where } \boldsymbol{\tau}_d = \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\mathbf{q}}_d} \right)^T - \left( \frac{\partial L}{\partial \mathbf{q}_d} \right)^T.\end{aligned}\tag{B.11}$$

## Appendix C

# Computation of Wrenches Reciprocal to a System of Twists

In this Appendix, we compute the actuation and constraint wrenches associated to some common *PKM* legs.

### C.1 Definitions

The twist  $\mathbf{t}$  of a body is parameterized by two vectors, the translational velocity  $\mathbf{v}$  and the rotational velocity  $\boldsymbol{\omega}$ , such that we can define a vector of dimension 6:

$$\mathbf{t} = \begin{bmatrix} \mathbf{v} \\ \boldsymbol{\omega} \end{bmatrix}. \quad (\text{C.1})$$

The twist  $\mathbf{t}$  is also called the velocity screw.  $\boldsymbol{\omega}$  is the resultant of the screw and  $\mathbf{v}$  is its moment.

A wrench  $\mathbf{w}$  is parameterized by two vectors, the force  $\mathbf{f}$  and the moment  $\mathbf{m}$ , such that we can define a vector of dimension 6:

$$\mathbf{w} = \begin{bmatrix} \mathbf{f} \\ \mathbf{m} \end{bmatrix}. \quad (\text{C.2})$$

The wrench  $\mathbf{w}$  is a screw in which  $\mathbf{f}$  is the resultant and  $\mathbf{m}$  is the moment of the screw.

Let us define the screws:

- $\$$  which describes a unit twist; as a result, if
  - $\$$  characterizes a pure translation, it can be written as

$$\$ = \begin{bmatrix} \mathbf{u}_1 \\ \mathbf{0} \end{bmatrix} \quad (\text{C.3})$$

in which  $\mathbf{u}_1$  represents the direction of the translation and  $\|\mathbf{u}_1\| = 1$ .

- $\$$  characterizes a pure rotation, it can be written as

$$\$ = \begin{bmatrix} \mathbf{u}_1 \times \mathbf{r} \\ \mathbf{u}_1 \end{bmatrix} \quad (\text{C.4})$$

in which  $\mathbf{u}_1$  represents the axis of the rotation ( $\|\mathbf{u}_1\| = 1$ ) and  $\mathbf{r}$  is the vector defining the distance between the axis of rotation and the point at which  $\$$  is expressed.

- $\zeta$  which describes a unit wrench.
  - $\zeta$  characterizes a pure moment, it can be written as

$$\zeta = \begin{bmatrix} \mathbf{0} \\ \mathbf{u}_2 \end{bmatrix} \quad (\text{C.5})$$

in which  $\mathbf{u}_2$  represents the direction around which the moment is applied and  $\|\mathbf{u}_2\| = 1$ .

- $\zeta$  characterizes a pure force, it can be written as

$$\zeta = \begin{bmatrix} \mathbf{u}_2 \\ \mathbf{u}_2 \times \mathbf{r} \end{bmatrix} \quad (\text{C.6})$$

in which  $\mathbf{u}_2$  represents the direction along which the force is applied ( $\|\mathbf{u}_2\| = 1$ ) and  $\mathbf{r}$  is the vector defining the distance between the point on which the force is applied and the point at which  $\zeta$  is expressed.

## C.2 Condition of Reciprocity

A twist  $\$$  is reciprocal to a wrench  $\zeta$  if their product is null, i.e.  $\$^T \zeta = \zeta^T \$ = 0$ . This means that the power developed by the wrench  $\zeta$  along the motion defined by  $\$$  is null.

From the definition of the unit twist and wrench  $\$$  and  $\zeta$ , we can define the following rules (Zhao et al. 2009):

- for a revolute joint with axis along the direction  $\mathbf{u}$ , the reciprocal wrenches are:
  - forces coplanar to  $\mathbf{u}$ , i.e. forces directed along an axis either parallel to  $\mathbf{u}$  or intersecting  $\mathbf{u}$  at a point,
  - moments whose axes are orthogonal to  $\mathbf{u}$ .
- for a prismatic joint with axis aligned along  $\mathbf{u}$ , the reciprocal wrenches are:
  - forces whose directions are orthogonal to  $\mathbf{u}$ ,
  - any moment.

These rules can be used to find the  $(6 - n)$  reciprocal wrenches for a system of  $n$  twists.

Similarly, we can define the twists reciprocal to a given wrench or system of wrenches. The interested reader could find more details in (Zhao et al. 2009).

### C.3 Computation of Wrenches Reciprocal to a System of Twists Constrained in a Plane

This case where a system of twists is constrained in a plane appear for the *PPM* (planar parallel manipulators). These mechanisms are made of joints whose displacements are all constrained in the same plane. The most common legs of *PPM* are presented in Fig. C.1. They are all made of three joints, one of them being actuated (the joints in gray).

Let us consider that the plane of motion is the  $(O_0, \mathbf{x}_0, \mathbf{y}_0)$  plane. As a result, any twist  $\$i$  associated to the motion of a joint  $i$  has the following form:

$$\$i = [v_{xi} \ v_{yi} \ 0 \ 0 \ 0 \ \omega_{zi}]^T. \quad (\text{C.7})$$

This means that, for any twist system of dimension  $n$  (representing a planar leg composed of  $n$  active or passive joints  $-n \in [1, +\infty[$ ) defined by  $\$ = [\$1 \ \dots \ \$n]$ , three constraint wrenches  $\zeta_{c1}$ ,  $\zeta_{c2}$  and  $\zeta_{c3}$  (i.e. the wrenches reciprocal to all twists representing the passive and active joint motions) can be easily defined as

$$\zeta_{c1} = [0 \ 0 \ 1 \ 0 \ 0 \ 0]^T \quad (\text{C.8})$$

which represents a force along  $\mathbf{z}_0$ ,

$$\zeta_{c2} = [0 \ 0 \ 0 \ 1 \ 0 \ 0]^T \quad (\text{C.9})$$

which represents a moment around  $\mathbf{x}_0$ , and

$$\zeta_{c3} = [0 \ 0 \ 0 \ 0 \ 1 \ 0]^T \quad (\text{C.10})$$

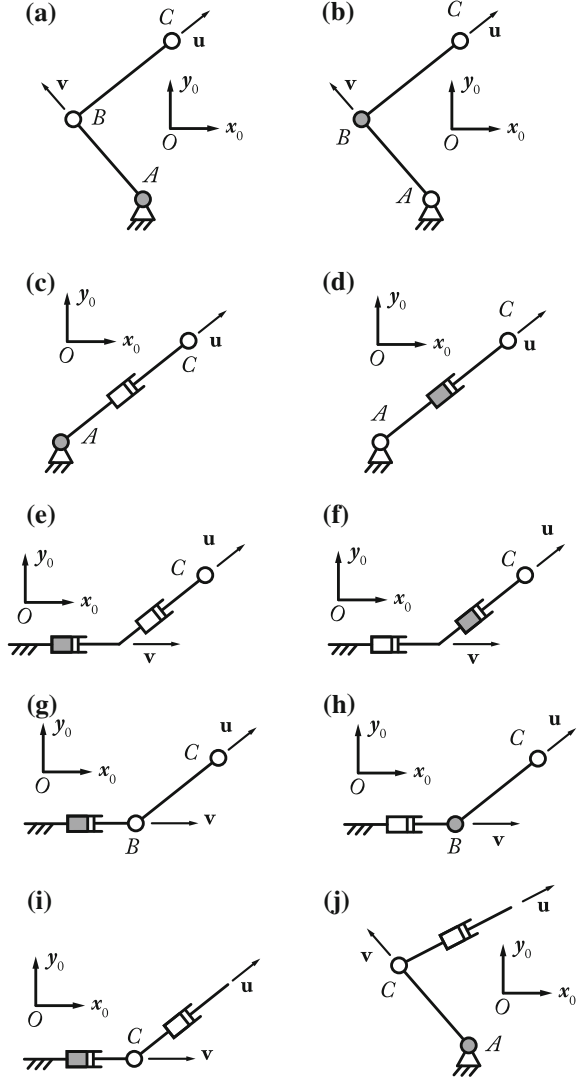
which represents a moment around  $\mathbf{y}_0$ .

These three constraint wrenches prevent the translation along  $\mathbf{z}_0$  and the rotations around  $\mathbf{x}_0$  and  $\mathbf{y}_0$  of the body located at the leg extremity. All legs presented in the Fig. C.1 impose these three constraint wrenches to the platform.

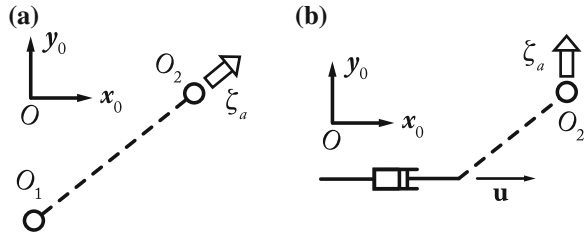
Now, let us consider that one joint of the legs depicted in Fig. C.1 is actuated (the joints in gray). This is the case for the most usual *PPM*. Starting from this consideration, only two cases can appear:

1. the passive system is made of two *R* joints (Fig. C.2a),
2. the passive system is made of one *P* joint and one *R* joint (Fig. C.2b).

**Fig. C.1** Usual legs for PPM. **a**  $\underline{R}\underline{R}\underline{R}$  leg. **b**  $\underline{R}\underline{R}\underline{R}$  leg. **c**  $\underline{R}\underline{P}\underline{R}$  leg. **d**  $\underline{R}\underline{P}\underline{R}$  leg. **e**  $\underline{P}\underline{P}\underline{R}$  leg. **f**  $\underline{P}\underline{P}\underline{R}$  leg. **g**  $\underline{P}\underline{P}\underline{R}$  leg. **h**  $\underline{P}\underline{P}\underline{R}$  leg. **i**  $\underline{P}\underline{R}\underline{P}$  leg. **j**  $\underline{R}\underline{R}\underline{R}$  leg



**Fig. C.2** General passive systems for the usual legs of PPM. **a**  $\underline{R}\underline{R}$  passive system ( $\zeta_a$  is aligned with  $(O_1 O_2)$ ). **b**  $\underline{P}\underline{R}$  passive system ( $\zeta_a \perp \underline{u}$ )



### ***C.3.1 Computation of Wrenches Reciprocal to a Twist System Representing the Motion of Two Passive R Joints***

For a system composed of two  $R$  joints (one located at point  $O_1$  and the second one at point  $O_2$ , point  $O_i$  having the coordinates  $(x_i, y_i)$  in the plane  $(O_0, \mathbf{x}_0, \mathbf{y}_0)$ —Fig. C.2a, it is possible to define two twists  $\$_{R1}$  and  $\$_{R2}$  parameterized by (when expressed at point  $O_2$ ):

$${}^0\$_{R1} = [-(y_2 - y_1) \ x_2 - x_1 \ 0 \ 0 \ 0 \ 1]^T \quad (\text{C.11})$$

$${}^0\$_{R2} = [0 \ 0 \ 0 \ 0 \ 0 \ 1]^T. \quad (\text{C.12})$$

The constraint wrenches  $\zeta_{ci}$  ( $i = 1, 2, 3$ ) are reciprocal to  $\$_{R1}$  and  $\$_{R2}$  but one additional wrench, denoted as the actuation wrench  $\zeta_a$  is reciprocal to these twists. It can be easily seen from (Zhao et al. 2009) that

$${}^0\zeta_a = [x_2 - x_1 \ y_2 - y_1 \ 0 \ 0 \ 0 \ 0]^T \quad (\text{C.13})$$

i.e. it is a force passing through the centers of the two  $R$  joints (see Fig. C.2a). It should be noted that  $\zeta_a$  must not be reciprocal to the twist of the third (actuated) joint

### ***C.3.2 Computation of Wrenches Reciprocal to a Twist System Representing the Motion of One P Joint and one R Joint***

For a system composed of  $P$  joint of direction  $\mathbf{u}_1 = [u_{1x} \ u_{1y} \ 0]^T$  and one  $R$  joint located at point  $O_2$  (Fig. C.2b), it is possible to define two twists  $\$_{P1}$  and  $\$_{R2}$  parameterized by (when expressed at point  $O_2$ ):

$${}^0\$_{P1} = [u_{1x} \ u_{1y} \ 0 \ 0 \ 0 \ 0]^T \quad (\text{C.14})$$

$${}^0\$_{R2} = [0 \ 0 \ 0 \ 0 \ 0 \ 1]^T. \quad (\text{C.15})$$

The constraint wrenches  $\zeta_{ci}$  ( $i = 1, 2, 3$ ) are reciprocal to  $\$_{P1}$  and  $\$_{R2}$  but one additional wrench, denoted as the actuation wrench  $\zeta_a$  is reciprocal to these twists ( $\zeta_a$  must not be reciprocal to the twist of the third (actuated) joint). It can be easily seen from (Zhao et al. 2009) that

$${}^0\zeta_a = [-u_{1y} \ u_{1x} \ 0 \ 0 \ 0 \ 0]^T \quad (\text{C.16})$$

i.e. it is a force lying in the plane  $(O_0, \mathbf{x}_0, \mathbf{y}_0)$  passing through  $O_2$  and orthogonal to the prismatic joint direction (see Fig. C.2b).

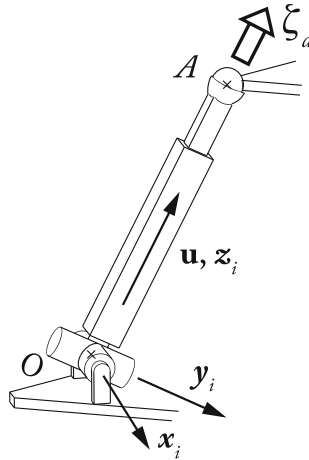
## C.4 Computation of Wrenches Reciprocal to Other Types of Twist Systems

For *SPM*, other types of twist systems appear. Due to the large number of possible leg architectures for the existing *SPM*, it is impossible to deal with all possible twist systems in this Appendix. However, we will compute the wrenches reciprocal to the twist systems corresponding to:

- *UPS* legs (legs of the Gough-Stewart platform—see Sect. 7.2.2.5)
- *UPU* legs (legs of the Tsai mechanism (Tsai and Joshi 2000)),
- *RUS* legs (legs of Hexa-like robots, and most of Delta-like robots (Clavel 1989; Company et al. 2002; Pashkevich et al. 2006)).

### C.4.1 Computation of Wrenches Reciprocal to a Twist System Representing the Motion of a *UPS* Leg

Let us consider a *UPS* leg composed of an actuated *P* joint of direction  ${}^i\mathbf{u} = [u_x \ 0 \ u_z]^T$  in the leg frame  $\mathcal{F}_i$  (Fig. C.3), one passive *U* joint which can be represented as an assembly of two *R* joints whose axes  $\mathbf{a}_1$  and  $\mathbf{a}_2$  are orthogonal to the direction of the *P* joint, i.e.  $\mathbf{a}_1 = [1 \ 0 \ 0]^T$  and  $\mathbf{a}_2 = [0 \ 1 \ 0]^T$  in the leg frame  $\mathcal{F}_i$ , and one passive *S* joint allowing three independent rotations around three axes  $\mathbf{a}_1 = [1 \ 0 \ 0]^T$ ,  $\mathbf{a}_2 = [0 \ 1 \ 0]^T$  and  $\mathbf{a}_3 = [0 \ 0 \ 1]^T$  without loss of generality in the leg frame  $\mathcal{F}_i$  (Fig. C.3).



**Fig. C.3** A *UPS* leg (in this configuration,  $\mathbf{u} \equiv \mathbf{z}_i$ , however, this is not the general case)



As a result, the twist system representing the motion of the leg in the frame  $\mathcal{F}_i$  and expressed at the center  $A$  of the  $S$  joint is given by:

$${}^i\mathbb{S}_1 = [0 \ -z_A \ 0 \ 1 \ 0 \ 0]^T \quad (\text{C.17})$$

$${}^i\mathbb{S}_2 = [z_A \ 0 \ -x_A \ 0 \ 1 \ 0]^T \quad (\text{C.18})$$

$${}^i\mathbb{S}_3 = [u_x \ 0 \ u_z \ 0 \ 0 \ 0]^T \quad (\text{C.19})$$

$${}^i\mathbb{S}_4 = [0 \ 0 \ 0 \ 1 \ 0 \ 0]^T \quad (\text{C.20})$$

$${}^i\mathbb{S}_5 = [0 \ 0 \ 0 \ 0 \ 1 \ 0]^T \quad (\text{C.21})$$

$${}^i\mathbb{S}_6 = [0 \ 0 \ 0 \ 0 \ 0 \ 1]^T \quad (\text{C.22})$$

where  $x_A$  and  $z_A$  are the coordinates of the point  $A$  along the axes  $\mathbf{x}_i$  and  $\mathbf{z}_i$  of the frame  $\mathcal{F}_i$ .

The total twist system  $\mathbb{S} = [\mathbb{S}_1 \ \dots \ \mathbb{S}_6]$  is of rank 6. Therefore, there are no constraint wrenches.

Now, let us compute the actuation wrench when the  $\underline{P}$  joint is considered actuated. We must thus consider the twist system  $\mathbb{S}_d = [\mathbb{S}_1 \ \mathbb{S}_2 \ \mathbb{S}_4 \ \mathbb{S}_5 \ \mathbb{S}_6]$ . It automatically gives that the actuation wrench  $\zeta_a$  is equal to, from (Zhao et al. 2009):

$${}^i\zeta_a = [u_x \ 0 \ u_z \ 0 \ 0 \ 0]^T \quad (\text{C.23})$$

i.e. it is a force directed along the direction connecting the centres of the  $U$  and  $S$  joints. We note that this force is not reciprocal to the actuated prismatic joint axis.

### ***C.4.2 Computation of Wrenches Reciprocal to a Twist System Representing the Motion of a $\underline{UPU}$ Leg***

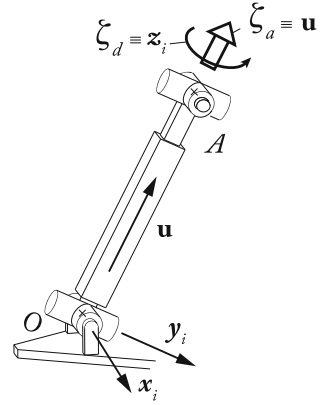
Let us consider a  $\underline{UPU}$  leg composed of an actuated  $\underline{P}$  joint of direction  $\mathbf{u} = [u_x \ 0 \ u_z]^T$  in the leg frame  $\mathcal{F}_i$ , and two passive  $U$  joints. Each passive  $U$  joint can be represented as an assembly of two  $R$  joints whose axes  $\mathbf{a}_1$  and  $\mathbf{a}_2$  are orthogonal to the direction of the  $\underline{P}$  joint, i.e.  $\mathbf{a}_1 = [1 \ 0 \ 0]^T$  and  $\mathbf{a}_2 = [0 \ 1 \ 0]^T$  in the leg frame  $\mathcal{F}_i$  (Fig. C.4).

As a result, the twist system representing the motion of the leg in the frame  $\mathcal{F}_i$  and expressed at the center  $A$  of the second  $U$  joint is given by:

$${}^i\mathbb{S}_1 = [0 \ -z_A \ y_A \ 1 \ 0 \ 0]^T \quad (\text{C.24})$$

$${}^i\mathbb{S}_2 = [z_A \ 0 \ -x_A \ 0 \ 1 \ 0]^T \quad (\text{C.25})$$

**Fig. C.4** A  $UPU$  leg (in this configuration,  $\mathbf{u} \equiv \mathbf{z}_i$ , however, this is not the general case)



$${}^i\mathcal{S}_3 = [u_x \ 0 \ u_z \ 0 \ 0 \ 0]^T \quad (\text{C.26})$$

$${}^i\mathcal{S}_4 = [0 \ 0 \ 0 \ 1 \ 0 \ 0]^T \quad (\text{C.27})$$

$${}^i\mathcal{S}_5 = [0 \ 0 \ 0 \ 0 \ 1 \ 0]^T. \quad (\text{C.28})$$

The total twist system  $\mathcal{S} = [\mathcal{S}_1 \dots \mathcal{S}_5]$  is of rank 5. As a result, there is a constraint wrench given from (Zhao et al. 2009) by

$${}^i\zeta_c = [0 \ 0 \ 0 \ 0 \ 0 \ 1]^T \quad (\text{C.29})$$

i.e. it is a moment around the  $\mathbf{z}_i$  axis (Fig. C.4).

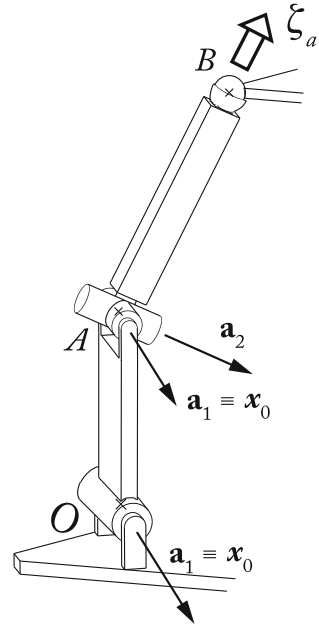
Now, let us compute the actuation wrench when the  $\underline{P}$  joint is considered actuated. We must thus consider the twist system  $\mathcal{S}_d = [\mathcal{S}_1 \ \mathcal{S}_2 \ \mathcal{S}_4 \ \mathcal{S}_5]$ . It automatically gives that the actuation wrench  $\zeta_a$  is equal to, from [Zhao et al., 2009]:

$${}^i\zeta_a = [u_x \ 0 \ u_z \ 0 \ 0 \ 0]^T \quad (\text{C.30})$$

i.e. it is a force directed along the direction of the prismatic joint (Fig. C.4).

### C.4.3 Computation of Wrenches Reciprocal to a Twist System Representing the Motion of a $\underline{RUS}$ Leg

Let us consider a  $\underline{RUS}$  leg composed of an actuated  $\underline{R}$  joint of direction  ${}^0\mathbf{a}_1 = [1 \ 0 \ 0]^T$  in the base frame  $\mathcal{F}_0$ , one passive  $\underline{U}$  joint which can be represented as an assembly of two  $\underline{R}$  joints whose axes are  $\mathbf{a}_1$  and  $\mathbf{a}_2$ , with  $\mathbf{a}_1 = [1 \ 0 \ 0]^T$  and  $\mathbf{a}_2 = [0 \ a_y \ a_z]^T$  in the leg frame  $\mathcal{F}_i$ , and one passive  $\underline{S}$  joint allowing three independent rotations around three axes  $\mathbf{a}_1 = [1 \ 0 \ 0]^T$ ,  $\mathbf{a}_3 = [0 \ 1 \ 0]^T$  and  $\mathbf{a}_4 = [0 \ 0 \ 1]^T$  without loss of generality in the leg frame  $\mathcal{F}_i$  (Fig. C.5).

**Fig. C.5** A  $\underline{RUS}$  leg

As a result, the twist system representing the motion of the leg in the frame  $\mathcal{F}_0$  and expressed at the center  $B$  of the  $S$  joint (with coordinates in the base frame  $x_B$ ,  $y_B$  and  $z_B$ ) is given by:

$${}^i\mathbb{S}_1 = [0 \ -z_B \ y_B \ 1 \ 0 \ 0]^T \quad (\text{C.31})$$

$${}^i\mathbb{S}_2 = [0 \ -z_{AB} \ y_{AB} \ 1 \ 0 \ 0]^T \quad (\text{C.32})$$

$${}^i\mathbb{S}_3 = [a_y z_{AB} - a_z y_{AB} \ a_z x_{AB} - a_y x_{AB} \ 0 \ a_y \ a_z]^T \quad (\text{C.33})$$

$${}^i\mathbb{S}_4 = [0 \ 0 \ 0 \ 1 \ 0 \ 0]^T \quad (\text{C.34})$$

$${}^i\mathbb{S}_5 = [0 \ 0 \ 0 \ 0 \ 1 \ 0]^T \quad (\text{C.35})$$

$${}^i\mathbb{S}_6 = [0 \ 0 \ 0 \ 0 \ 0 \ 1]^T \quad (\text{C.36})$$

where  $x_{AB}$ ,  $y_{AB}$  and  $z_{AB}$  are the coordinates of the vector  $\overrightarrow{AB}$  along the axes of the frame  $\mathcal{F}_0$ . Note that  $\overrightarrow{AB} \perp \mathbf{a}_2$ , i.e. that  $y_{AB}a_y + z_{AB}a_z = 0$ .

The total twist system  $\mathbb{S} = [\mathbb{S}_1 \dots \mathbb{S}_6]$  is of rank 6. Therefore, there are no constraint wrenches.

Now, let us compute the actuation wrench when the first  $R$  joint is considered actuated. We must thus consider the twist system  $\$d = [\$2 \dots \$6]$ . It automatically gives that the actuation wrench  $\xi_a$  is equal to, from (Zhao et al. 2009):

$${}^i\xi_a = [x_{AB} \ y_{AB} \ z_{AB} \ 0 \ 0 \ 0]^T / \sqrt{x_{AB}^2 + y_{AB}^2 + z_{AB}^2} \quad (\text{C.37})$$

i.e. it is a force directed along the direction given by the vector  $\overrightarrow{AB}$  (Fig. C.5). We can verify that  $\xi_a$  is not orthogonal to the unit twist of the actuated joint.

## Appendix D

### Point-to-Point Trajectory Generation

Let us consider a robot displacement between an initial configuration  $A_0$  parameterized by the Cartesian variables  $\mathbf{x}_0$  and a final configuration  $A_f$  parameterized by the Cartesian variables  $\mathbf{x}_f$ . The trajectory between these two configurations can be defined by the functions

$$\mathbf{x}(t) = s(t) (\mathbf{x}_f - \mathbf{x}_0) + \mathbf{x}_0 \quad (\text{D.1})$$

$$\dot{\mathbf{x}}(t) = \dot{s}(t) (\mathbf{x}_f - \mathbf{x}_0) \quad (\text{D.2})$$

$$\ddot{\mathbf{x}}(t) = \ddot{s}(t) (\mathbf{x}_f - \mathbf{x}_0) \quad (\text{D.3})$$

where:

- $t \in [0, t_f]$ , where  $t = 0$  s is the time at which the robot starts to move from the initial configuration  $A_0$  and  $t_f$  is the time at which the robot arrives at the final configuration  $A_f$
- $\mathbf{x}(t)$  denotes the robot Cartesian variables at the time  $t$ ,
- $\dot{\mathbf{x}}(t)$  denotes the first derivative w.r.t. time of the robot Cartesian variables at the time  $t$ ,
- $\ddot{\mathbf{x}}(t)$  denotes the second derivative w.r.t. time of the robot Cartesian variables at the time  $t$ ,
- $s(t)$  is an interpolation function.

We can deduce from (D.1) that the path in the Cartesian space will be defined by a straight line. The boundary conditions for  $s(t)$  are deduced as:

$$s(t = 0 \text{ s}) = 0 \quad (\text{D.4})$$

$$s(t = t_f) = 1. \quad (\text{D.5})$$

Moreover, if at the initial and final configurations, the velocities and accelerations are null, we have

$$\dot{s}(t = 0 \text{ s}) = 0 \quad (\text{D.6})$$

$$\dot{s}(t = t_f) = 0 \quad (\text{D.7})$$

$$\ddot{s}(t = 0 \text{ s}) = 0 \quad (\text{D.8})$$

$$\ddot{s}(t = t_f) = 0. \quad (\text{D.9})$$

From these boundary conditions, and assuming that the interpolation function  $s(t)$  is a polynomial of the form:

$$s(t) = \sum_{k=0}^n a_k t^k \quad (\text{D.10})$$

we can find the coefficients  $a_k$ .

For high speed robots or when a robot is handling heavy or delicate loads, it is worth ensuring the continuity of the position, velocity, and accelerations as well, in order to avoid exciting resonances in the mechanics. The trajectory is said to be of class  $C^2$ . Thus, from (D.2) and (D.3), we must define the functions  $\dot{s}(t)$  and  $\ddot{s}(t)$ , given by

$$\dot{s}(t) = \sum_{k=1}^n k a_k t^{k-1} \quad (\text{D.11})$$

and

$$\ddot{s}(t) = \sum_{k=2}^n k(k-1) a_k t^{k-2}. \quad (\text{D.12})$$

Since six constraints (D.4)–(D.9) have to be satisfied, the interpolation requires a polynomial of at least fifth degree (Binford et al. 1977).

Solving the six constraints yields the following interpolation function:

$$s(t) = 10 \left( \frac{t}{t_f} \right)^3 - 6 \left( \frac{t}{t_f} \right)^4 + 15 \left( \frac{t}{t_f} \right)^5. \quad (\text{D.13})$$

Obviously, if we increase the number of boundary conditions to take into account, the order of the polynomial will increase. For example, if  $n_c$  constraints have to be satisfied, the interpolation requires a polynomial of at least  $n_c - 1$  degree.

## Appendix E

### Calculation of the Terms $\mathbf{f}_{acc1}$ , $\mathbf{f}_{acc2}$ and $\mathbf{f}_{acc3}$ in Chapter 10

#### E.1 Calculation of the Term $\mathbf{f}_{acc1}$

From (10.12) and (10.15), we get that  $\mathbf{f}_{acc1}$  is given by

$$\begin{aligned}\mathbf{f}_{acc1} &= \int_{\mathcal{B}_j} \dot{\mathbf{v}}_{M_j} dm \\ &= \int_{\mathcal{B}_j} (\dot{\mathbf{v}}_j + \Phi_{d_j}(M_{0j}) \ddot{\mathbf{q}}_{e_j} + 2\boldsymbol{\omega}_j \times \Phi_{d_j}(M_{0j}) \dot{\mathbf{q}}_{e_j}) dm \\ &\quad + \int_{\mathcal{B}_j} (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j}) + \dot{\boldsymbol{\omega}}_j \times \mathbf{r}_{O_j M_j}) dm\end{aligned}\quad (\text{E.1})$$

which can be expanded to be rewritten as

$$\begin{aligned}\mathbf{f}_{acc1} &= \int_{\mathcal{B}_j} \dot{\mathbf{v}}_j dm + \int_{\mathcal{B}_j} \Phi_{d_j}(M_{0j}) dm \ddot{\mathbf{q}}_{e_j} \\ &\quad + \dot{\boldsymbol{\omega}}_j \times \left( \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} dm \right) \\ &\quad + 2\boldsymbol{\omega}_j \times \left( \int_{\mathcal{B}_j} \Phi_{d_j}(M_{0j}) dm \dot{\mathbf{q}}_{e_j} \right) \\ &\quad + \boldsymbol{\omega}_j \times \left( \boldsymbol{\omega}_j \times \left( \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} dm \right) \right).\end{aligned}\quad (\text{E.2})$$

#### E.2 Calculation of the Term $\mathbf{f}_{acc2}$

From (10.12) and (10.15), we get that  $\mathbf{f}_{acc2}$  is given by

$$\begin{aligned}
\mathbf{f}_{acc2} &= \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} \times \dot{\mathbf{v}}_{M_j} dm \\
&= \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} \times (\dot{\mathbf{v}}_j + \Phi_{d_j}(M_{0j})\ddot{\mathbf{q}}_{e_j} + 2\boldsymbol{\omega}_j \times \Phi_{d_j}(M_{0j})\dot{\mathbf{q}}_{e_j}) dm \\
&\quad + \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} \times (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j}) + \dot{\boldsymbol{\omega}}_j \times \mathbf{r}_{O_j M_j}) dm \quad (E.3)
\end{aligned}$$

which can be expanded to be rewritten under a sum of five terms:

$$\mathbf{f}_{acc2} = \sum_{k=1}^5 \mathbf{a}_k \quad (E.4)$$

where:

$$\begin{aligned}
\mathbf{a}_1 &= \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} \times \dot{\mathbf{v}}_j dm \quad (E.5) \\
&= \left( \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} dm \right) \times \dot{\mathbf{v}}_j
\end{aligned}$$

$$\begin{aligned}
\mathbf{a}_2 &= \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} \times (\Phi_{d_j}(M_{0j})\ddot{\mathbf{q}}_{e_j}) dm \\
&= \int_{\mathcal{B}_j} \hat{\mathbf{r}}_{O_j M_j} \Phi_{d_j}(M_{0j})\ddot{\mathbf{q}}_{e_j} dm \quad (E.6) \\
&= \left( \int_{\mathcal{B}_j} \hat{\mathbf{r}}_{O_j M_j} \Phi_{d_j}(M_{0j}) dm \right) \ddot{\mathbf{q}}_{e_j}
\end{aligned}$$

$$\begin{aligned}
\mathbf{a}_3 &= \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} \times (\dot{\boldsymbol{\omega}}_j \times \mathbf{r}_{O_j M_j}) dm \\
&= - \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} \times (\mathbf{r}_{O_j M_j} \times \dot{\boldsymbol{\omega}}_j) dm \\
&= - \int_{\mathcal{B}_j} \hat{\mathbf{r}}_{O_j M_j} (\hat{\mathbf{r}}_{O_j M_j} \dot{\boldsymbol{\omega}}_j) dm \quad (E.7) \\
&= \int_{\mathcal{B}_j} \hat{\mathbf{r}}_{O_j M_j}^T \hat{\mathbf{r}}_{O_j M_j} \dot{\boldsymbol{\omega}}_j dm \\
&= \int_{\mathcal{B}_j} \hat{\mathbf{r}}_{O_j M_j}^T \hat{\mathbf{r}}_{O_j M_j} dm \dot{\boldsymbol{\omega}}_j
\end{aligned}$$

$$\mathbf{a}_4 = \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} \times (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j})) dm \quad (E.8)$$

$$\mathbf{a}_5 = 2 \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_j} \times (\boldsymbol{\omega}_j \times \Phi_{d_j}(M_{0j})\dot{\mathbf{q}}_{e_j}) dm. \quad (E.9)$$



To simplify the term  $\mathbf{a}_4$ , let us recall the well-known identity for the double cross-product of three arbitrary vectors  $\mathbf{u}$ ,  $\mathbf{v}$  and  $\mathbf{w}$ :

$$\mathbf{u} \times (\mathbf{v} \times \mathbf{w}) + \mathbf{w} \times (\mathbf{u} \times \mathbf{v}) + \mathbf{v} \times (\mathbf{w} \times \mathbf{u}) = \mathbf{0}. \quad (\text{E.10})$$

If  $\mathbf{u} = \mathbf{r}_{O_j M_j}$ ,  $\mathbf{v} = \boldsymbol{\omega}_j$ ,  $\mathbf{w} = \boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j}$ , so  $\mathbf{w} \times (\mathbf{u} \times \mathbf{v}) = \mathbf{0}$  which leads to

$$\mathbf{u} \times (\mathbf{v} \times \mathbf{w}) = -\mathbf{v} \times (\mathbf{w} \times \mathbf{u}) \quad (\text{E.11})$$

or also, replacing the vectors  $\mathbf{u}$ ,  $\mathbf{v}$  and  $\mathbf{w}$  by their corresponding values

$$\mathbf{r}_{O_j M_j} \times (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j})) = -\boldsymbol{\omega}_j \times ((\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j}) \times \mathbf{r}_{O_j M_j}). \quad (\text{E.12})$$

Thus,  $\mathbf{a}_4$  becomes

$$\begin{aligned} \mathbf{a}_4 &= - \int_{\mathcal{B}_j} \boldsymbol{\omega}_j \times ((\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j}) \times \mathbf{r}_{O_j M_j}) \, dm \\ &= - \int_{\mathcal{B}_j} \boldsymbol{\omega}_j \times (\mathbf{r}_{O_j M_j} \times (\mathbf{r}_{O_j M_j} \times \boldsymbol{\omega}_j)) \, dm \\ &= - \int_{\mathcal{B}_j} \boldsymbol{\omega}_j \times (\hat{\mathbf{r}}_{O_j M_j} \hat{\mathbf{r}}_{O_j M_j} \boldsymbol{\omega}_j) \, dm \\ &= \boldsymbol{\omega}_j \times \left( \left( \int_{\mathcal{B}_j} \hat{\mathbf{r}}_{O_j M_j}^T \hat{\mathbf{r}}_{O_j M_j} \, dm \right) \boldsymbol{\omega}_j \right). \end{aligned} \quad (\text{E.13})$$

Now, introducing (10.4) into (E.9), the expression of  $\mathbf{a}_5$  becomes

$$\mathbf{a}_5 = 2(\mathbf{a}_{51} + \mathbf{a}_{52}) \quad (\text{E.14})$$

where

$$\begin{aligned} \mathbf{a}_{51} &= \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_{0j}} \times (\boldsymbol{\omega}_j \times \Phi_{d_j}(M_{0j}) \dot{\mathbf{q}}_{e_j}) \, dm \\ &= - \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_{0j}} \times ((\Phi_{d_j}(M_{0j}) \dot{\mathbf{q}}_{e_j}) \times \boldsymbol{\omega}_j) \, dm \end{aligned} \quad (\text{E.15})$$

$$\begin{aligned} \mathbf{a}_{52} &= \int_{\mathcal{B}_j} (\Phi_{d_j}(M_{0j}) \mathbf{q}_{e_j}) \times (\boldsymbol{\omega}_j \times \Phi_{d_j}(M_{0j}) \dot{\mathbf{q}}_{e_j}) \, dm \\ &= - \int_{\mathcal{B}_j} (\Phi_{d_j}(M_{0j}) \mathbf{q}_{e_j}) \times ((\Phi_{d_j}(M_{0j}) \dot{\mathbf{q}}_{e_j}) \times \boldsymbol{\omega}_j) \, dm. \end{aligned} \quad (\text{E.16})$$

In order to simplify these two expressions, let us consider the fact that:

$$\Phi_{dj}(M_{0j})\mathbf{q}_{ej} = \sum_{k=1}^{N_j} \Phi_{dkj}(M_{0j})q_{ekj} \quad (\text{E.17})$$

$$\Phi_{dj}(M_{0j})\dot{\mathbf{q}}_{ej} = \sum_{k=1}^{N_j} \Phi_{dkj}(M_{0j})\dot{q}_{ekj}. \quad (\text{E.18})$$

Introducing (E.18) into (E.15) leads to:

$$\begin{aligned} \mathbf{a}_{51} &= - \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_{0j}} \times \left( \left( \sum_{k=1}^{N_j} \Phi_{dkj}(M_{0j})\dot{q}_{ekj} \right) \times \boldsymbol{\omega}_j \right) dm \\ &= - \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_{0j}} \times ((\Phi_{dkj}(M_{0j})\dot{q}_{ekj}) \times \boldsymbol{\omega}_j) dm \right) \\ &= - \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \mathbf{r}_{O_j M_{0j}} \times (\Phi_{dkj}(M_{0j}) \times \boldsymbol{\omega}_j) dm \dot{q}_{ekj} \right) \\ &= \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \hat{\mathbf{r}}_{O_j M_{0j}}^T \hat{\Phi}_{dkj}(M_{0j}) dm \right) \boldsymbol{\omega}_j \dot{q}_{ekj}. \end{aligned} \quad (\text{E.19})$$

Then, introducing (E.17) and (E.18) into (E.16) leads to:

$$\begin{aligned} \mathbf{a}_{52} &= - \int_{\mathcal{B}_j} \left( \sum_{i=1}^{N_j} \Phi_{di_j}(M_{0j})q_{ei_j} \right) \times \left( \left( \sum_{k=1}^{N_j} \Phi_{dkj}(M_{0j})\dot{q}_{ekj} \right) \times \boldsymbol{\omega}_j \right) dm \\ &= - \sum_{i=1}^{N_j} \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} (\Phi_{di_j}(M_{0j})q_{ei_j}) \times ((\Phi_{dkj}(M_{0j})\dot{q}_{ekj}) \times \boldsymbol{\omega}_j) dm \right) \\ &= - \sum_{i=1}^{N_j} \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \Phi_{di_j}(M_{0j}) \times (\Phi_{dkj}(M_{0j}) \times \boldsymbol{\omega}_j) dm q_{ei_j} \dot{q}_{ekj} \right) \\ &= - \sum_{i=1}^{N_j} \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \hat{\Phi}_{di_j}(M_{0j}) \hat{\Phi}_{dkj}(M_{0j}) \boldsymbol{\omega}_j dm q_{ei_j} \dot{q}_{ekj} \right) \\ &= \sum_{i=1}^{N_j} \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \hat{\Phi}_{di_j}^T(M_{0j}) \hat{\Phi}_{dkj}(M_{0j}) dm \right) \boldsymbol{\omega}_j q_{ei_j} \dot{q}_{ekj}. \end{aligned} \quad (\text{E.20})$$

Finally, introducing expressions (E.5), (E.6), (E.7), (E.13), (E.14), (E.19), and (E.20) into (E.4), the expression (10.18) can be obtained.

### E.3 Calculation of the Term $\mathbf{f}_{acc3}$

From (10.12) and (10.15), we get that  $\mathbf{f}_{acc3}$  is given by

$$\begin{aligned}\mathbf{f}_{acc3} &= \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) \dot{\mathbf{v}}_{M_j} dm \\ &= \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\dot{\mathbf{v}}_j + \Phi_{d_j}(M_{0j}) \ddot{\mathbf{q}}_{e_j} + 2\boldsymbol{\omega}_j \times \Phi_{d_j}(M_{0j}) \dot{\mathbf{q}}_{e_j}) dm \\ &\quad + \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j}) + \dot{\boldsymbol{\omega}}_j \times \mathbf{r}_{O_j M_j}) dm\end{aligned}\quad (\text{E.21})$$

which can be expanded to be rewritten under a sum of five terms:

$$\mathbf{f}_{acc3} = \sum_{k=1}^5 \mathbf{b}_k \quad (\text{E.22})$$

where:

$$\mathbf{b}_1 = \int_{\mathcal{B}_j} \Phi_{d_j}(M_{0j})^T \dot{\mathbf{v}}_j dm \quad (\text{E.23})$$

$$\begin{aligned}\mathbf{b}_2 &= \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\Phi_{d_j}(M_{0j}) \ddot{\mathbf{q}}_{e_j}) dm \\ &= \left( \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) \Phi_{d_j}(M_{0j}) dm \right)^T \ddot{\mathbf{q}}_{e_j}\end{aligned}\quad (\text{E.24})$$

$$\begin{aligned}\mathbf{b}_3 &= \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\dot{\boldsymbol{\omega}}_j \times \mathbf{r}_{O_j M_j}) dm \\ &= - \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\mathbf{r}_{O_j M_j} \times \dot{\boldsymbol{\omega}}_j) dm \\ &= - \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) \hat{\mathbf{r}}_{O_j M_j} \dot{\boldsymbol{\omega}}_j dm \\ &= \left( \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) \hat{\mathbf{r}}_{O_j M_j}^T dm \right) \dot{\boldsymbol{\omega}}_j \\ &= \left( \int_{\mathcal{B}_j} \hat{\mathbf{r}}_{O_j M_j} \Phi_{d_j}(M_{0j}) dm \right)^T \dot{\boldsymbol{\omega}}_j\end{aligned}\quad (\text{E.25})$$

$$\mathbf{b}_4 = \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j})) dm \quad (\text{E.26})$$

$$\mathbf{b}_5 = 2 \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times \Phi_{d_j}(M_{0j}) \dot{\mathbf{q}}_{e_j}) dm. \quad (\text{E.27})$$

Considering the  $i$ th component  $\mathbf{b}_4|_i$  of the vector  $\mathbf{b}_4$ , we can rewrite (E.26) as

$$\mathbf{b}_4|_i = \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_j})) dm \quad (\text{E.28})$$

in which  $\Phi_{d_j}(M_{0j})$  is the  $i$ th column of the matrix  $\Phi_{d_j}(M_{0j})$ .

Now, introducing (10.4) into (E.28), the expression of  $\mathbf{b}_4|_i$  becomes

$$\mathbf{b}_4|_i = b_{4i1} + b_{4i2} \quad (\text{E.29})$$

where

$$b_{4i1} = \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_{0j}})) dm \quad (\text{E.30})$$

$$= \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) ((\mathbf{r}_{O_j M_{0j}} \times \boldsymbol{\omega}_j) \times \boldsymbol{\omega}_j) dm$$

$$b_{4i2} = \int_{\mathcal{B}_j} \Phi_{d_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times (\Phi_{d_j} \mathbf{q}_{e_j}))) dm. \quad (\text{E.31})$$

To simplify the term  $b_{4i1}$ , let us recall the well-known identity for the triple product of three arbitrary vectors  $\mathbf{u}$ ,  $\mathbf{v}$  and  $\mathbf{w}$ :

$$\mathbf{u}^T (\mathbf{v} \times \mathbf{w}) = \mathbf{w}^T (\mathbf{u} \times \mathbf{v}). \quad (\text{E.32})$$

Now, replacing  $\mathbf{u}$  by  $\Phi_{d_j}(M_{0j})$ ,  $\mathbf{v}$  by  $\boldsymbol{\omega}_j \times \mathbf{r}_{O_j M_{0j}}$ , and  $\mathbf{w}$  by  $\boldsymbol{\omega}_j$ , we get

$$\begin{aligned} b_{4i1} &= \int_{\mathcal{B}_j} \boldsymbol{\omega}_j^T (\Phi_{d_j}(M_{0j}) \times (\mathbf{r}_{O_j M_{0j}} \times \boldsymbol{\omega}_j)) dm \\ &= \boldsymbol{\omega}_j^T \left( \int_{\mathcal{B}_j} \hat{\Phi}_{d_j}(M_{0j}) \hat{\mathbf{r}}_{O_j M_{0j}} dm \right) \boldsymbol{\omega}_j \\ &= -\boldsymbol{\omega}_j^T \left( \int_{\mathcal{B}_j} \hat{\Phi}_{d_j}^T(M_{0j}) \hat{\mathbf{r}}_{O_j M_{0j}} dm \right) \boldsymbol{\omega}_j \\ &= -\boldsymbol{\omega}_j^T \left( \int_{\mathcal{B}_j} \hat{\mathbf{r}}_{O_j M_{0j}}^T \hat{\Phi}_{d_j}(M_{0j}) dm \right)^T \boldsymbol{\omega}_j. \end{aligned} \quad (\text{E.33})$$

Now, in order to simplify the term  $b_{4i2}$ , let us consider the fact that:

$$\Phi_{dj}(M_{0j})\mathbf{q}_{ej} = \sum_{k=1}^{N_j} \Phi_{dkj}(M_{0j})q_{ekj}. \quad (\text{E.34})$$

Thus,  $b_{4i2}$  can be rewritten as

$$\begin{aligned} b_{4i2} &= \int_{\mathcal{B}_j} \Phi_{di_j}^T(M_{0j}) \left( \boldsymbol{\omega}_j \times \left( \boldsymbol{\omega}_j \times \left( \sum_{k=1}^{N_j} \Phi_{dkj}(M_{0j})q_{ekj} \right) \right) \right) dm \\ &= \sum_{k=1}^{N_j} \int_{\mathcal{B}_j} \Phi_{di_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \Phi_{dkj}(M_{0j})q_{ekj})) dm \\ &= \sum_{k=1}^{N_j} \int_{\mathcal{B}_j} \Phi_{di_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times (\boldsymbol{\omega}_j \times \Phi_{dkj}(M_{0j}))) dm q_{ekj}. \end{aligned} \quad (\text{E.35})$$

Once again, using the identity (E.32) by replacing  $\mathbf{u}$  by  $\boldsymbol{\omega}_j$ ,  $\mathbf{v}$  by  $\boldsymbol{\omega}_j \times \Phi_{dkj}(M_{0j})$ , and  $\mathbf{w}$  by  $\Phi_{di_j}(M_{0j})$ , we can obtain

$$\begin{aligned} b_{4i2} &= \sum_{k=1}^{N_j} \int_{\mathcal{B}_j} \boldsymbol{\omega}_j^T ((\boldsymbol{\omega}_j \times \Phi_{dkj}(M_{0j})) \times \Phi_{di_j}(M_{0j})) dm q_{ekj} \\ &= \sum_{k=1}^{N_j} \boldsymbol{\omega}_j^T \left( \int_{\mathcal{B}_j} \Phi_{di_j}(M_{0j}) \times (\Phi_{dkj}(M_{0j}) \times \boldsymbol{\omega}_j) dm \right) q_{ekj} \\ &= \sum_{k=1}^{N_j} \boldsymbol{\omega}_j^T \left( \int_{\mathcal{B}_j} \hat{\Phi}_{di_j}(M_{0j}) \hat{\Phi}_{dkj}(M_{0j}) \boldsymbol{\omega}_j dm \right) q_{ekj} \\ &= - \sum_{k=1}^{N_j} \boldsymbol{\omega}_j^T \left( \int_{\mathcal{B}_j} \hat{\Phi}_{di_j}^T(M_{0j}) \hat{\Phi}_{dkj}(M_{0j}) dm \right) \boldsymbol{\omega}_j q_{ekj}. \end{aligned} \quad (\text{E.36})$$

Now, considering the  $i$ th component  $\mathbf{b}_5|_i$  of the vector  $\mathbf{b}_5$ , we can rewrite (E.27) as

$$\mathbf{b}_5|_i = 2 \int_{\mathcal{B}_j} \Phi_{di_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times \Phi_{dj}(M_{0j})\dot{\mathbf{q}}_{ej}) dm. \quad (\text{E.37})$$

In order to simplify the term  $\mathbf{b}_5|_i$ , let us consider the fact that:

$$\Phi_{dj}(M_{0j})\dot{\mathbf{q}}_{ej} = \sum_{k=1}^{N_j} \Phi_{dkj}(M_{0j})\dot{q}_{ekj}. \quad (\text{E.38})$$

Thus,  $\mathbf{b}_5|_i$  can be rewritten as

$$\begin{aligned}
 \mathbf{b}_5|_i &= 2 \int_{\mathcal{B}_j} \Phi_{di_j}^T(M_{0j}) \left( \boldsymbol{\omega}_j \times \left( \sum_{k=1}^{N_j} \Phi_{dk_j}(M_{0j}) \dot{q}_{ek_j} \right) \right) dm \\
 &= 2 \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \Phi_{di_j}^T(M_{0j}) (\boldsymbol{\omega}_j \times \Phi_{dk_j}(M_{0j})) dm \dot{q}_{ek_j} \right) \quad (\text{E.39}) \\
 &= -2 \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \Phi_{di_j}^T(M_{0j}) (\Phi_{dk_j}(M_{0j}) \times \boldsymbol{\omega}_j) dm \dot{q}_{ek_j} \right).
 \end{aligned}$$

Using the identity (E.32) by replacing  $\mathbf{u}$  by  $\Phi_{di_j}(M_{0j})$ ,  $\mathbf{v}$  by  $\boldsymbol{\omega}_j$ , and  $\mathbf{w}$  by  $\Phi_{dk_j}(M_{0j})$ , we can obtain

$$\begin{aligned}
 \mathbf{b}_5|_i &= -2 \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \Phi_{di_j}^T(M_{0j}) (\Phi_{dk_j}(M_{0j}) \times \boldsymbol{\omega}_j) dm \dot{q}_{ek_j} \right) \\
 &= -2 \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \boldsymbol{\omega}_j^T (\Phi_{di_j}^T(M_{0j}) \times \Phi_{dk_j}(M_{0j})) dm \dot{q}_{ek_j} \right) \\
 &= 2 \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \boldsymbol{\omega}_j^T (\Phi_{dk_j}(M_{0j}) \times \Phi_{di_j}^T(M_{0j})) dm \dot{q}_{ek_j} \right) \quad (\text{E.40}) \\
 &= 2 \sum_{k=1}^{N_j} \left( \int_{\mathcal{B}_j} \Phi_{dk_j}(M_{0j}) \times \Phi_{di_j}^T(M_{0j}) dm \right)^T \boldsymbol{\omega}_j \dot{q}_{ek_j}.
 \end{aligned}$$

Finally, taking the  $i$ th rows of expressions (E.23), (E.24) and (E.25) and summing them with the expressions (E.33), (E.36) and (E.40), the expression (10.19) can be obtained.

## Appendix F

### Dynamics Equations for a Clamped-Free Flexible Beam

#### F.1 Shape Functions for a Free Flexible Beam

Computation of the mass and stiffness matrices of 3D beams is useful for elastodynamic modeling of parallel manipulators.

The Bernoulli model describes beam deformation under the assumption that the shear effect is negligible, that the cross-sections remain perpendicular to the neutral axis and that the rotational inertia of sections is assumed to be zero Blevins (2001). With such a model, the 3D beam deformation  $\mathbf{u}_{ej}(M_{0j})$  (see Sect. 10.2.1) can be characterized with the six shape functions  $\Phi_{dxj}$ ,  $\Phi_{dyj}$ ,  $\Phi_{dzj}$ ,  $\Phi_{rxj}$ ,  $\Phi_{ryj}$  and  $\Phi_{rzj}$ , i.e.  $N_j = 6$ , defined as:

$$\Phi_{dxj} = [\xi \ 0 \ 0 \ 0 \ 0 \ 0] \quad (\text{F.1a})$$

$$\Phi_{dyj} = [0 \ 3\xi^2 - 2\xi^3 \ 0 \ 0 \ 0 \ l_j (\xi^3 - \xi^2)] \quad (\text{F.1b})$$

$$\Phi_{dzj} = [0 \ 0 \ 3\xi^2 - 2\xi^3 \ 0 \ -l_j (\xi^3 - \xi^2) \ 0] \quad (\text{F.1c})$$

$$\Phi_{rxj} = [0 \ 0 \ 0 \ \xi \ 0 \ 0] \quad (\text{F.1d})$$

$$\Phi_{ryj} = [0 \ 0 \ -6(\xi - \xi^2) / l_j \ 0 \ 3\xi^2 - 2\xi \ 0] \quad (\text{F.1e})$$

$$\Phi_{rzj} = [0 \ 6(\xi - \xi^2) / l_j \ 0 \ 0 \ 0 \ 3\xi^2 - 2\xi] \quad (\text{F.1f})$$

where  $\xi = x/l_j$  and  $l_j$  is the beam length.

$x$ ,  $y$  and  $z$  denote the Cartesian coordinates of point  $M_{0j}$  expressed in the local frame  $\mathcal{F}_j$  and  $\Phi_{dj}(M_{0j})$  defined at (10.3) is a  $(3 \times 6)$  matrix that takes the form:

$$\Phi_{dj}(M_{0j}) = \begin{bmatrix} \Phi_{dxj} - y\Phi_{rzj} + z\Phi_{ryj} \\ \Phi_{dyj} - z\Phi_{rxj} \\ \Phi_{dzj} + y\Phi_{rxj} \end{bmatrix} \quad (\text{F.2})$$

while  $\Phi_{rj}(M_{0j})$  defined at (10.5) is a  $(3 \times 6)$  matrix equal to:

$$\Phi_{rj}(M_{0j}) = \begin{bmatrix} \Phi_{rxj} \\ \Phi_{ryj} \\ \Phi_{rzj} \end{bmatrix}. \quad (\text{F.3})$$

## F.2 Stiffness Matrix for a Free Flexible Beam

In the beam model, it is assumed that (Shabana 2005)

$$\sigma_{j22} = \sigma_{j33} = \sigma_{j23} = 0 \quad (\text{F.4})$$

$$\varepsilon_{j22} = \varepsilon_{j33} = \varepsilon_{j23} = 0 \quad (\text{F.5})$$

$$\sigma_{j11} = E_j \varepsilon_{j11} \quad (\text{F.6})$$

$$\sigma_{j12} = G_j \varepsilon_{j12} \quad (\text{F.7})$$

$$\sigma_{j13} = G_j \varepsilon_{j13} \quad (\text{F.8})$$

where  $E_j$  is the Young modulus of body  $j$  and  $G_j = E_j/(2(1 + \nu_j))$  is its shear modulus,  $\nu_j$  being the Poisson's coefficient.

Introducing (F.1a) to (F.8) into (10.35), the stiffness matrix of body  $\mathcal{B}_j$  takes the form:

$$\mathbf{K}_{eej} = \frac{1}{l_j^3} \begin{bmatrix} E_j A_j l_j^2 & 0 & 0 & 0 & 0 & 0 \\ 0 & 12E_j I_{zj} & 0 & 0 & 0 & -6E_j I_{zj} l_j \\ 0 & 0 & 12E_j I_{yj} & 0 & 6E_j I_{yj} l_j & 0 \\ 0 & 0 & 0 & I_{0j} G_j l_j^2 & 0 & 0 \\ 0 & 0 & 6E_j I_{yj} l_j & 0 & 4E_j I_{yj} l_j^2 & 0 \\ 0 & -6E_j I_{zj} l_j & 0 & 0 & 0 & 4E_j I_{zj} l_j^2 \end{bmatrix} \quad (\text{F.9})$$

where  $A_j$  is the beam cross-section area,  $I_{yj}$  and  $I_{zj}$  are the second moments of area around axes  $y$  and  $z$  of the local frame,  $I_{0j}$  is the torsion constant.

## F.3 Evaluation of the Inertia Matrix of a Free Flexible 3D Bernoulli Beam for $\mathbf{q}_{ej} = \mathbf{0}$

For  $\mathbf{q}_{ej} = \mathbf{0}$ , the inertia matrix of the flexible 3D Bernoulli beam becomes, from Sect. 10.2.24:

$$\mathbf{M}_j = \begin{bmatrix} m_j \mathbf{1}_3 & \widehat{\mathbf{m}} \mathbf{s}_{rj}^T & \mathbf{M} \mathbf{S}_{dej} \\ \widehat{\mathbf{m}} \mathbf{s}_{rj} & \mathbf{I}_{rrj} & \mathbf{M} \mathbf{S}_{rej}^r \\ \mathbf{M} \mathbf{S}_{dej}^T & \mathbf{M} \mathbf{S}_{rej}^{rT} & \mathbf{M}_{eej} \end{bmatrix} \quad (\text{F.10})$$



where

$$\mathbf{MS}_{rej}^r = [\beta_{1j} \cdots \beta_{Nj,j}]. \quad (\text{F.11})$$

After simplifications, we get that

$$\mathbf{MS}_{dej} = \begin{bmatrix} \frac{m_j}{2} & 0 & 0 & 0 & 0 & 0 \\ 0 & \frac{m_j}{2} & 0 & 0 & 0 & \frac{m_j l_j}{12} \\ 0 & 0 & \frac{m_j}{2} & 0 & \frac{m_j l_j}{12} & 0 \end{bmatrix} \quad (\text{F.12})$$

$$\mathbf{MS}_{rej}^r = \begin{bmatrix} 0 & 0 & 0 & \frac{\rho_j l_j I_{pj}}{2} & 0 & 0 \\ 0 & 0 & -\rho_j I_{y_j} - \frac{7m_j l_j}{20} & 0 & -\frac{m_j l_j^2}{20} & 0 \\ 0 & \rho_j I_{z_j} + \frac{7m_j l_j}{20} & 0 & 0 & 0 & -\frac{m_j l_j^2}{20} \end{bmatrix} \quad (\text{F.13})$$

and

$$\mathbf{M}_{ej} = \begin{bmatrix} \frac{m_j}{3} & 0 & 0 & 0 & 0 & 0 \\ 0 & \frac{13m_j}{35} + \frac{6\rho_j I_{z_j}}{5l_j} & 0 & 0 & 0 & -\frac{11m_j l_j + 21\rho_j I_{z_j}}{210} \\ 0 & 0 & \frac{13m_j}{35} + \frac{6\rho_j I_{y_j}}{5l_j} & 0 & \frac{11m_j l_j + 21\rho_j I_{y_j}}{210} & 0 \\ 0 & 0 & 0 & \frac{\rho_j l_j I_{pj}}{3} & 0 & 0 \\ 0 & 0 & \frac{11m_j l_j + 21\rho_j I_{y_j}}{210} & 0 & \frac{m_j l_j^2 + 14\rho_j I_{y_j} l_j}{105} & 0 \\ 0 & -\frac{11m_j l_j + 21\rho_j I_{z_j}}{210} & 0 & 0 & 0 & \frac{m_j l_j^2 + 14\rho_j I_{z_j} l_j}{105} \end{bmatrix} \quad (\text{F.14})$$

where  $I_{pj} = I_{y_j} + I_{z_j}$  is the polar moment of inertia.

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